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(54) Title: BLIND RIVET AND FASTENING ARRANGEMENT WITH A BLIND RIVET

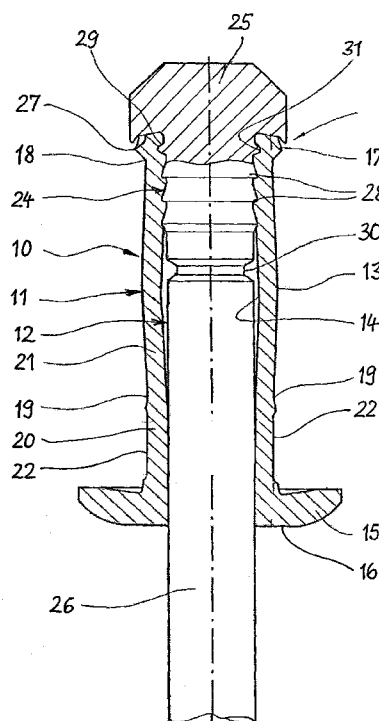


FIG. 1

(57) Abstract: A blind rivet (10) for joining components made of plastic has a hollow rivet body (11) with an elongated shank (13) and a flange (15) and, located in the bore of the rivet body (11), a mandrel (12) with a mandrel shank (24), a drawing end (26), and a mandrel head (25). The shank (13) of the rivet body (11) has a first section (20) adjoining the flange (15) and a second section (21) adjacent thereto. The second section (21) is designed to form a flange that lies flat against the component located on the inaccessible side. The strength of the first section (20) is increased with respect to the strength of the second section (21) in such a manner that the outside diameter of the first section (20) does not increase or increases only slightly during setting of the blind rivet (10).



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BLIND RIVET AND FASTENING ARRANGEMENT WITH A BLIND RIVET

The invention relates to a blind rivet, in particular for joining components made of plastic, having a hollow rivet body that has an elongated shank that
5 extends coaxially to a longitudinal center axis of the blind rivet and has a bore, having a flange extending radially at one end of the shank for contact with an accessible side of a component, and having, at the opposite end of the shank, a foot end that is passed through openings in the components, and having, located in the bore of the rivet body, a mandrel that has a mandrel shank, a
10 drawing end projecting out of the flange of the rivet body that can be separated from the mandrel shank, and a mandrel head that stands in operative connection with the foot end of the rivet body in order to transmit a force to the rivet body during setting of the blind rivet. The invention additionally relates to a fastening arrangement with such a blind rivet.

15 A blind rivet of the specified type is known from WO 2009/098431 A1. Such blind rivets are used to permanently join workpieces that are in contact with one another. The workpieces typically have bores or holes passing through them that are brought into alignment with one another and into which the blind rivet is placed and then fastened by deformation. A fastened blind
20 rivet usually results in a clearance-free seating of the rivet body in the holes of the workpieces and clamps the workpieces together.

In the automotive industry, uses of blind rivets include fastening attached parts such as power-window motors, loudspeakers, and other parts, to a component support, such as a door module support, using one or more
25 blind rivets. In many cases, the mounting portions of the attached parts and component supports are made of plastic, with the result that the fastening with blind rivets was only satisfactory with the use of additional components made of metal to reinforce the plastic parts. Without metal reinforcing parts, setting of the rivet joints can occur that could lead to a detachment of the joint under
30 load. The settings are caused by the customary formation of the blind head during setting of the blind rivet. Furthermore, under certain circumstances the

mounting hole in the plastic component may split due to the radial expansion of the rivet shank during setting of the rivet.

The object of the invention is to create a blind rivet of the initially mentioned type and a fastening arrangement with such a blind rivet that permit
5 improved joining of components made of plastic.

This object is achieved according to the invention by a blind rivet with the features specified in claim 1, and a fastening arrangement with the features specified in claim 11. Advantageous refinements of the invention are specified in the other claims.

10 According to the invention, the blind rivet comprises a hollow rivet body that has an elongated shank that extends coaxially to a longitudinal center axis of the blind rivet and has a bore, a flange extending radially at one end of the shank for contact with an accessible side of a component, and, at the opposite
15 end of the shank, a foot end that is passed through openings in the components, and additionally has, located in the bore of the rivet body, a mandrel that has a mandrel shank, a drawing end that projects out of the flange of the rivet body and can be separated from the mandrel shank, and a mandrel head that stands in operative connection with the foot end of the rivet
20 body in order to transmit a force to the rivet body during setting of the blind rivet, wherein the shank of the rivet body has a first section adjoining the flange and, adjacent thereto and extending to the foot end, a second section, and the second section is designed such that, as a result of a process in which the flange is pressed against one side of the components and the foot end is
25 simultaneously drawn toward the other side of the components with the aid of the mandrel, a flange forms that lies flat against the component located on the inaccessible side, and wherein the strength of the first section of the shank is increased with respect to the strength of the second section in such a manner that the outside diameter of the first section does not increase or increases only slightly as a result of the process.

30 The blind rivet according to the invention is distinguished from prior art blind rivets in that the first section of the shank of the rivet body adjoining the flange is hardened in such a manner that it is not expanded or is expanded

only slightly during setting of the blind rivet. Consequently, the hardened section of the blind rivet can be placed in a component hole with a slight oversize without having an increased radial pressure arise between the rivet section and the wall of the bore during setting, which could cause damage to the component. The blind rivet according to the invention is thus suitable for fastening components made of lower strength materials such as plastic, with it being possible to dispense with additional metal reinforcing parts, such as washers or tubular rivets. The elimination of additional reinforcing parts made of metal also results in cost advantages during assembly, and expands the range of application possibilities for the blind rivet according to the invention.

According to another proposal of the invention, the increase in strength of the first section of the rivet body is accomplished through plastic deformation. The plastic deformation can be accomplished in that the first section has multiple radial indentations, compressions, or crimps, which extend over a substantial part of its length. As a result of the plastic deformation, the first section may also be provided with multiple longitudinal ribs. The plastic deformation produces work hardening in the region of the first section, causing the deformation resistance of the first section to be significantly increased as compared to the second section. The first section can thus better withstand the forces occurring during deformation of the second section of the rivet body, so that no radial deformation, or at most minor radial deformation, of the first section occurs during setting of the blind rivet. Additionally, the strengthening of the first section by plastic deformation has the advantage that the diameter of the first section can correspond essentially to the diameter of the second section, and the rivet body is easy to manufacture. The foot end of the rivet body also can have an increased strength brought about by plastic deformation so that it can better withstand compressive forces acting during the setting process.

According to another proposal of the invention, provision can be made for the mandrel shank to have at least one constriction and/or projection in the region of the foot end and for the foot end to be brought by plastic deformation into a shape that engages in the constriction and/or engages around the

projection. By this means, the rivet body is joined to the rivet mandrel in an interlocking manner so that the two parts form an easy-to-manage structural unit. During setting of the blind rivet, the mandrel shank with the attached mandrel head is held securely by the rivet body as a result of this interlocking connection. By this means, disadvantages resulting from a released mandrel head and mandrel shank are avoided. The constriction and/or projection can be designed in an annular form and can accomplish an improved sealing of the mandrel shank with respect to the rivet body in conjunction with the plastically deformed foot end.

10 According to the invention, the mandrel shank and the drawing end are joined by a constricted parting section, which is arranged in the bore of the rivet body such that the end of the mandrel shank adjacent to the parting section is recessed within the bore of the rivet body. The danger of injury by the sometimes sharp-edged broken end of the mandrel shank is avoided by
15 this means.

A fastening arrangement with at least two components that are joined together by the setting of a blind rivet according to the invention comprises an accessible first component and a second component located inaccessibly behind the first component, wherein the two components have bores that are
20 aligned with one another through which the blind rivet passes, wherein the flange of the blind rivet rests against the first component, the first section of the rivet's shank extends through the first component, and the second section of the shank forms a flange that lies flat against the back of the second component, and wherein the outside diameter of the first section of the shank
25 is equal to or smaller than the original inside diameter of the bore in the first component.

In the fastening arrangement according to the invention, the two components are clamped together between the flange and the second section that has been formed into a flange. However, the expansion of the first section
30 of the blind rivet is reduced enough that its outside diameter essentially does not increase beyond the inside diameter of the bore in the first component. Overstressing of the component through expansion of the blind rivet during

setting is avoided, so that splitting of the component also cannot occur. Consequently, the first component can be made of plastic and does not require any reinforcements made of metal.

If both components are made of plastic, it is advantageous for the length of the first section of the shank to be essentially equal to the sum of the thicknesses of the first and second components in the region of the bores that the blind rivet passes through, or to be smaller than the said sum. The reduction in the expansion of the blind rivet is then effective for the second component as well.

For tight sealing of the fastening arrangement, according to the invention the mandrel shank is enclosed in an interlocking manner by the foot end of the rivet body, by which means it forms a seal together with the mandrel head for the blind-side end of the bore of the rivet body.

The invention is explained in detail below with reference to exemplary embodiments that are shown in the drawings, wherein:

Figure 1 is a cross-section of a blind rivet according to the invention, and

Figure 2 is a partial section of a fastening arrangement according to the invention.

Figure 1 shows a blind rivet 10, which is composed of a rivet body 11 made of aluminum and a mandrel 12 made of steel. The rivet body 11 is attached to the mandrel 12 and has an elongated shank 13 with a through-bore 14, in which the mandrel is located. Formed at a head end of the shank 13 is a flange 15, which takes the form of an annular disk and is intended to contact a workpiece. The side of the flange 15 facing away from the shank 13 is provided with a flat support surface 16, whose purpose is supporting the forward end of a rivet setting tool.

At its end opposite the flange 15, the shank 13 forms a foot end 17, which is offset from the shank 13 by means of a first constriction 18. A second

constriction 19 divides the shank 13 into an essentially cylindrical first section 20, and a barrel-like second section 21 that curves slightly outward. On its outside, the first section 20 is provided with multiple flat indentations 22 spaced at regular intervals from one another; these extend in the axial
5 direction over almost the entire length of the first section 20. As a result of the indentations 22 produced by way of cold forming, the strength of the shank 13 in the region of the first section 20 is increased with the result that the first section 20 is no longer expanded or is only slightly expanded during setting of the blind rivet 10. In the region of the first section 20 and in the region of the
10 foot end 17, the rivet body 11 is in contact with the mandrel 12; in the central region that curves outward, an annular clearance is present between the rivet body 11 and the mandrel 12.

The mandrel 12 has a mandrel shank 24 that bears a mandrel head 25 at one end, and an elongated drawing end 26 at the other end. The mandrel
15 head 25 has a greater diameter than the mandrel shank 24, and has, on the underside facing the mandrel shank 24, flat contact surfaces 27 for the foot end 17 of the shank 13. The contact surfaces 27 are inclined at an acute angle of approximately 70° to 80° with respect to the longitudinal axis of the mandrel 12. In place of the flat contact surfaces 27, a concave conical surface
20 with corresponding inclination may also be provided. The end of the mandrel head 25 opposite the contact surfaces 27 is in the shape of a truncated cone in order to facilitate insertion of the blind rivet 10 in the workpiece bores.

On part of its length, the mandrel shank 24 has locking means in the form of annular ribs 28, which are spaced apart from one another and have a
25 sawtooth-like cross-section that is composed of conical surfaces and radial surfaces. The conical surfaces of the ribs 28 face the mandrel head 25. Located between the mandrel head 25 and the rib 28 adjacent thereto is a rib 29, which is embedded in a groove 31 in the foot end 17 and holds the mandrel 12 in place in the rivet body 11 against the recoil that attempts to drive
30 the mandrel 12 out of the rivet body 11 when the drawing end 26 pulls off during setting of the blind rivet. The rib 29 also ensures an effective seal between the rivet body 11 and the mandrel 12 after the blind rivet has been

set. The bore 14 is adapted to the outside contour of the ribs 28, 29 by radial deformation of the rivet body, forming an interlocking connection between the mandrel shank 25 and the shank 13, which contributes to the transmission of tensile forces to the rivet body during setting of the blind rivet. Moreover, the
5 mandrel 12 is held fast in the bore 14 of the shank 13, so that the rivet body 11 and the mandrel 12 compose an easy-to-manipulate unit.

The drawing end 26 is connected to the mandrel shank 24 by means of a constricted parting section 30 that forms a predetermined breaking point. The parting section is located in the bore 14 of the rivet body 11 and has a
10 spacing from the flange 15 such that the breaking point on the mandrel shank 24 is located in the bore 14 even after setting of the blind rivet 10.

Figure 2 shows a fastening arrangement with two components 32, 33 joined by a blind rivet 10. The components 32, 33 have bores 34, 35. For the
15 purpose of joining, the blind rivet 10 is inserted in the aligned bores 34, 35 of the components 32, 33. With the aid of a setting tool that bears on the support surface 16 of the blind rivet 10, a tensile force is then exerted on the drawing end 26 of the mandrel 12, and the rivet body is deformed, causing the foot end 17 to approach the flange 15. During this process, the barrel-like second
20 section 21 on the blind side of the components 32, 33 folds to form a flange 36 that lies flat against the component 33 and presses the components 32, 33 against one another and against the flange 15. Consequently, the plastic components do not require any reinforcement by additional components made of metal, since the flange 36 rests virtually flat against the component 33. The
25 component 33 experiences only a reduced radial load, because the flange 36 forms as a disk shape rather than as a cone shape. The cylindrical section 20 is located in the bores 34, 35 and is loaded in compression by the tensile force that is exerted. The increased strength of the section 20 that has been produced by the indentations 22 prevents or reduces a radial expansion in the
30 section 20, however, so that no radial pressure, or at most a slight radial pressure, takes place between the section 20 and the walls of the bores 34, 35. As a result, the stress on the components 32, 33 in the radial direction is

low, and the risk of overloading or damaging the components during setting of the blind rivet is avoided. Only in the blind-side end region of the bore 35 in the component 33 may a limited radial pressure resulting from radial expansion of the shank 13 be desirable in order to achieve a better seal between the

5 blind rivet and component 33.

CLAIMS

1. Blind rivet for joining at least two components, in particular for joining components made of plastic, having a hollow rivet body that has an elongated shank that extends coaxially to a longitudinal center axis of the blind rivet and has a bore, a flange extending radially at one end of the shank for contact with an accessible side of a component, and, at the opposite end of the shank, a foot end that is passed through openings in the components, and having, located in the bore of the rivet body, a mandrel that has a mandrel shank, a drawing end projecting out of the flange of the rivet body that can be separated from the mandrel shank, and a mandrel head that stands in operative connection with the foot end of the rivet body in order to transmit a force to the rivet body during setting of the blind rivet, characterized in that the shank (13) of the rivet body (11) has a first section (20) adjoining the flange (15) and, adjacent thereto and extending to the foot end (17), a second section (21), wherein the second section (21) is designed such that, as a result of a process in which the flange (15) is pressed against one side of the components and the foot end (19) is simultaneously drawn toward the other side of the components with the aid of the mandrel (12), a flange (36) forms that lies flat against the component located on the inaccessible side, and wherein the strength of the first section (20) of the shank (13) is increased with respect to the strength of the second section (21) in such a manner that the outside diameter of the first section (20) does not increase or increases only slightly as a result of the process.

25

2. Blind rivet according to claim 1, characterized in that the first section (20) is plastically deformed to increase its strength.

3. Blind rivet according to one of claims 1 or 2, characterized in that the first section (20) has multiple radial indentations (22), compressions, or crimps, which extend over a substantial part of its length.

30

4. Blind rivet according to one of claims 2 or 3, characterized in that the first section has multiple longitudinal ribs formed by plastic deformation.

5. Blind rivet according to one of the preceding claims,
5 characterized in that the foot end (17) of the shank (13) has an increased strength as compared to the second section (21).

6. Blind rivet according to one of the preceding claims,
characterized in that the second section (21) of the shank (13) has a barrel-like
10 shape that curves outward.

7. Blind rivet according to one of the preceding claims,
characterized in that the mandrel shank (24) has, in the region of the foot end
(17), at least one constriction and/or projection and in that the foot end is
15 brought by plastic deformation into a shape that engages in the constriction
and/or engages around the projection.

8. Blind rivet according to one of the preceding claims,
characterized in that the mandrel shank (24) and the drawing end (26) are
20 joined by a constricted parting section (30), which is located in the bore (14) of
the rivet body (11).

9. Blind rivet according to one of the preceding claims,
characterized in that the rivet head (25) has a contact surface (27) for
25 supporting the foot end (17) that is inclined at an acute angle with respect to
the longitudinal axis of the mandrel (12).

10. Blind rivet according to one of the preceding claims,
characterized in that the rivet body (11) is made of aluminum and the mandrel
30 (12) is made of steel.

11. Fastening arrangement with at least two components that are joined by setting of a blind rivet according to one of the preceding claims, characterized by an accessible first component (32) and a second component (33) located inaccessibly behind the first component (32), wherein the two components (32, 33) have bores (34, 45) that are aligned with one another through which the blind rivet (10) passes, wherein the flange (15) of the blind rivet (10) rests against the first component (32), the first section (20) of the rivet's shank (13) extends through the first component (32), and the second section (21) of the shank (13) forms a flange (36) that lies flat against the back of the second component (33), and wherein the outside diameter of the first section (20) is essentially equal to or smaller than the original inside diameter of the bore (34) in the accessible, first component (32).

12. Fastening arrangement according to claim 11, characterized in that the length of the first section (20) of the shank is essentially equal to the sum of the thicknesses of the joined components (32, 33) in the region of the bores (34, 35) that the blind rivet (10) passes through, or is smaller than the said sum.

13. Fastening arrangement according to one of claims 11 or 12, characterized in that the foot end (17) of the rivet body (11) encloses the mandrel shank (24) and holds it in the bore (14) of the rivet body (11) in an interlocking manner.

14. Fastening arrangement according to claim 13, characterized in that the end of the mandrel shank (24) separated from the drawing end (26) is recessed within the bore (14) of the rivet body (11).

15. Fastening arrangement according to one of claims 11 through 14, characterized in that the accessible, first component (32) is made of plastic.

16. Fastening arrangement according to one of claims 11 through 15, characterized in that both components (32, 33) are made of plastic.

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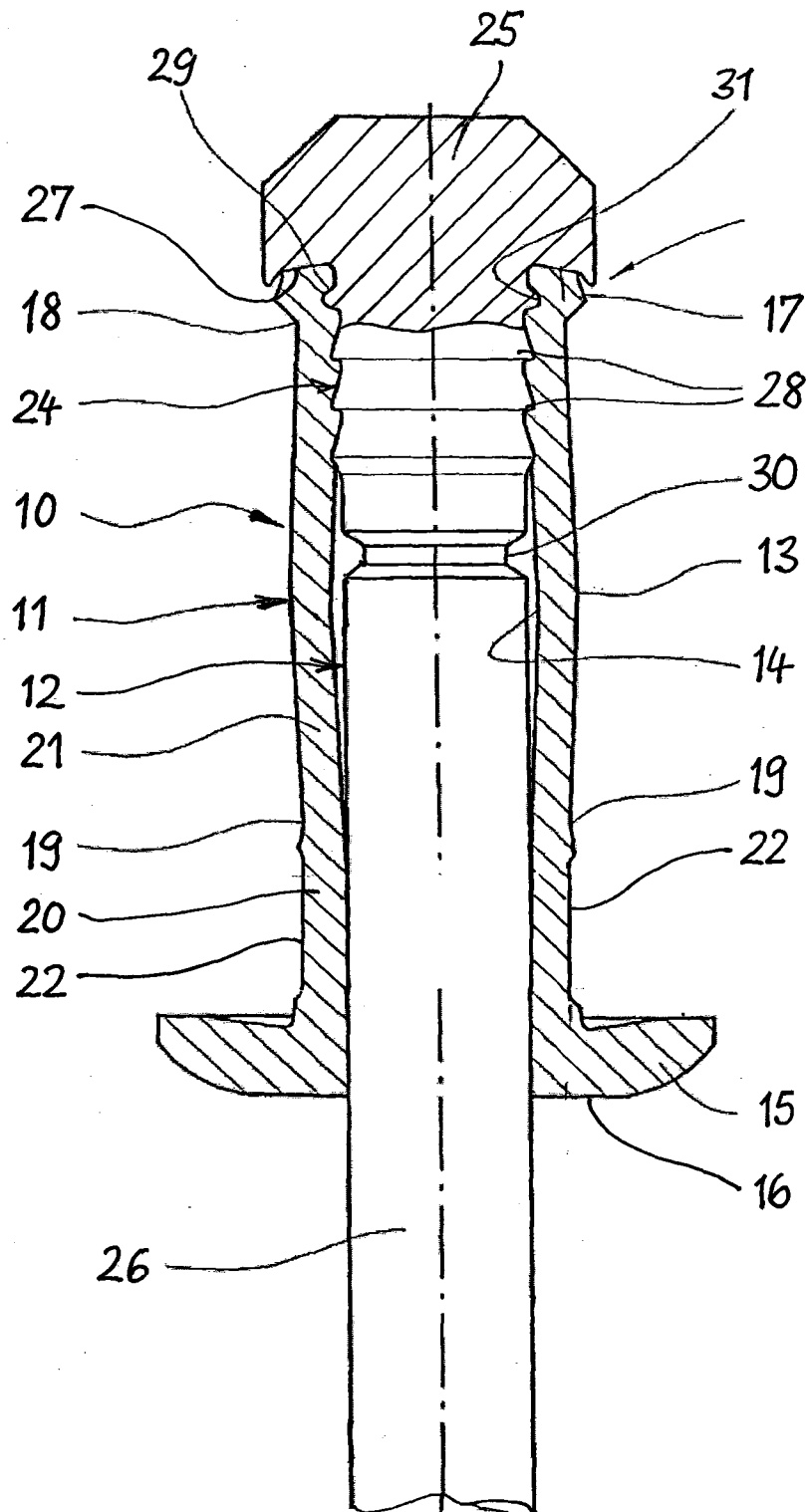


FIG. 1

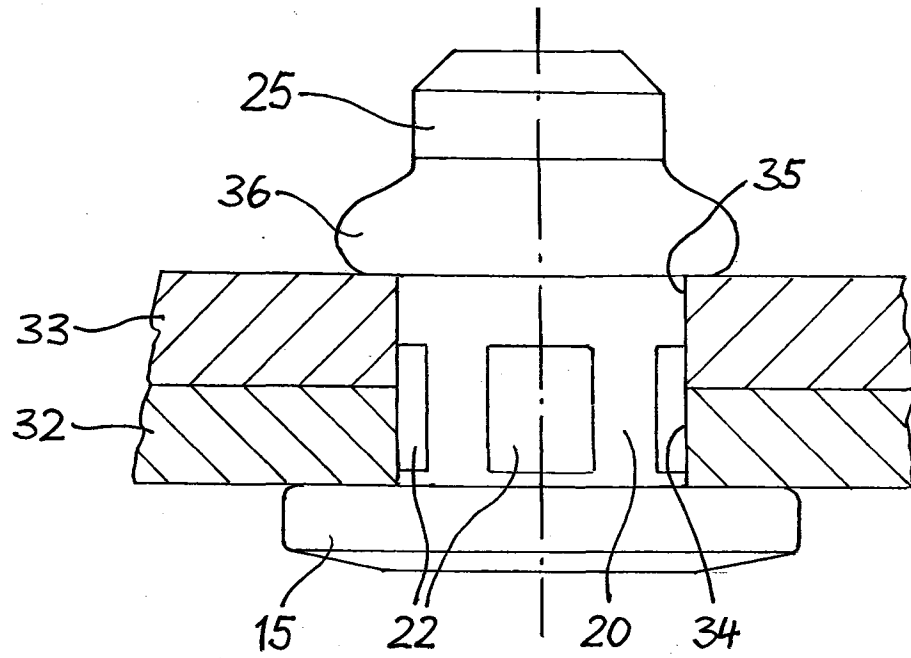


FIG. 2

INTERNATIONAL SEARCH REPORT

International application No

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A. CLASSIFICATION OF SUBJECT MATTER

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ADD.

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

F16B

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

EPO-Internal

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	GB 1 046 239 A (HUCK MFG CO) 19 October 1966 (1966-10-19)	1,8,11, 12
Y	page 1, line 26 - page 3, line 11 figures 1-3	2,3,15, 16
Y	----- EP 1 217 229 A2 (EMHART LLC [US] NEWFREY LLC [US]) 26 June 2002 (2002-06-26)	2,3,15, 16
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A	----- EP 1 030 069 A2 (EMHART INC [US]) 23 August 2000 (2000-08-23) the whole document	1-16
A	----- US 3 426 375 A (JEAL HARVEY PHILIP) 11 February 1969 (1969-02-11) figures 10,11	1-16



Further documents are listed in the continuation of Box C.



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INTERNATIONAL SEARCH REPORT

Information on patent family members

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