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(54) **ROTARY COMPRESSOR**

ROTATIONSVERDICHTER

COMPRESSEUR ROTATIF

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Description

Technical Field

[0001] The present disclosure relates to a rotary compressor.

[0002] This application claims the priority of Japanese Patent Application No. 2020-065992 filed in Japan on April 1, 2020.

Background Art

[0003] A rotary compressor is known as a device used for compressing a refrigerant in an air conditioner. The rotary compressor includes a motor, a shaft driven by the motor, a rotary piston attached to the shaft, and a cylinder covering the rotary piston. The refrigerant is compressed by an eccentric rotation of the rotary piston in the compression chamber of the cylinder.

[0004] In recent years, as the above-mentioned motor, a type called an axial gap motor has been widely used (for example, PTL 1 below). An axial gap motor described in PTL 1 has one stator and two rotors facing the stator from both sides in an axial direction. The rotary piston and the cylinder described above are independently disposed below the axial gap motor.

Citation List

Patent Literature

[0005] [PTL 1] Japanese Unexamined Patent Application Publication No. 2008-106694

Summary of Invention

[0006] [PTL 2] CN 104 564 685 A.

Technical Problem

[0007] However, when the axial gap motor and the cylinder are disposed independently as described above, there are problems that the number of parts increases and a size of the device increases.

[0008] The present disclosure has been made in order to solve the above problems, and an object of the present disclosure is to provide a rotary compressor having a reduced number of parts and a smaller size. Solution to Problem

[0009] In order to solve the above problems, according to the inventive idea of the present disclosure, there is provided a rotary compressor as defined by the accompanying claims, including: a shaft that extends along an axis; a disk-shaped rotor that is fixed to the shaft and centered on the axis; a pair of stators that faces the rotor from both sides in a direction of the axis, and includes disk-shaped back yokes centered on the axis, teeth protruding from the back yoke, and a coil wound around

the teeth; a pair of cylinders that abuts on the stator from the direction of the axis and has an annular shape centered on the axis; rotary pistons that rotate eccentrically with the shaft; a pair of end plates that forms each a compression chamber for accommodating a rotary piston together with a back yoke by sandwiching the cylinder together with a stator from the direction of the axis; and a pair of bearings that is provided on at least one of the end plate and the back yoke. Advantageous Effects of Invention

[0010] According to the present disclosure, it is possible to provide a rotary compressor having a reduced number of parts and a smaller size.

15 Brief Description of Drawings

[0011]

Fig. 1 is a vertical sectional view of a rotary compressor according to a first embodiment of the present disclosure.

Fig. 2 is an enlarged cross-sectional view of a main part of the rotary compressor according to the first embodiment of the present disclosure.

Fig. 3 is an enlarged cross-sectional view of a main part of a rotary compressor according to a second embodiment of the present disclosure.

Fig. 4 is an enlarged cross-sectional view of a main part showing a modification example of the rotary compressor according to the second embodiment of the present disclosure. Description of Embodiments

[First Embodiment]

35 (Configuration of Rotary Compressor)

[0012] Hereinafter, a rotary compressor 100 according to a first embodiment of the present disclosure will be described with reference to Figs. 1 and 2. As shown in Fig. 1, the rotary compressor 100 according to the present embodiment includes a shaft 1, a rotor 2, a stator 3, a cylinder 4, an end plate 5, a bearing 6, a rotary piston 12, and a housing 7 for accommodating these.

[0013] The shaft 1 has a shaft main body 1H, an upper eccentric shaft 11A, and a lower eccentric shaft 11B. The shaft main body 1H has a columnar shape extending along an axis Ac. The upper eccentric shaft 11A and the lower eccentric shaft 11B are provided at an interval in a direction of the axis Ac. Each of the upper eccentric shaft 11A and the lower eccentric shaft 11B has a disk shape eccentric in a radial direction with respect to the axis Ac. Eccentric directions of the upper eccentric shaft 11A and the lower eccentric shaft 11B are different from each other. For example, the eccentric direction of the upper eccentric shaft 11A differs from the eccentric direction of the lower eccentric shaft 11B by 180°.

[0014] The rotor 2 is integrally provided at a position (central portion) in a middle of extension of the shaft main

body 1H. That is, the rotor 2 is provided at an intermediate position between the upper eccentric shaft 11A and the lower eccentric shaft 11B. The rotor 2 has a rotor core 21 and a permanent magnet 22. The rotor core 21 has a disk shape centered on the axis Ac. The permanent magnet 22 has a ring shape extending along a peripheral edge of the rotor core 21. Instead of the permanent magnet 22, it is possible to adopt a configuration in which a plurality of magnets are arranged on the peripheral edge of the rotor core 21 at intervals in a circumferential direction.

[0015] The stator 3 disposed so as to face the rotor 2 from both sides in the direction of the axis Ac includes an upper stator 3A and a lower stator 3B. The upper stator 3A faces the rotor 2 from one side (upper side) in the direction of the axis Ac. The upper stator 3A has a back yoke 31A, teeth 32A, and a coil 33A. The back yoke 31A has an annular shape centered on the axis Ac. An opening through which the shaft 1 is inserted is formed in a portion including the center of the back yoke 31A. The teeth 32A are positioned on a surface of the back yoke 31A facing the other side (lower side) in the direction of the axis Ac, and have a rod shape protruding in the direction of the axis Ac from the center position in the radial direction. A plurality of teeth 32A are arranged at equal intervals in the circumferential direction with respect to the axis Ac. The coil 33A is formed by winding a copper wire around each tooth 32A. Power is supplied to the coil 33A from a power source (not shown).

[0016] The lower stator 3B has a back yoke 31B, teeth 32B, and a coil 33B. The back yoke 31B has an annular shape centered on the axis Ac. An opening through which the shaft 1 is inserted is formed in a portion including the center of the back yoke 31B. The teeth 32B are positioned on a surface of the back yoke 31B facing one side (upper side) in the direction of the axis Ac, and have a rod shape protruding in the direction of the axis Ac from the center position in the radial direction. A plurality of teeth 32B are arranged at equal intervals in the circumferential direction with respect to the axis Ac. The coil 33B is formed by winding a copper wire around each tooth 32B. Power is supplied to the coil 33B from a power source (not shown). As a result, the upper stator 3A and the lower stator 3B are excited, and the shaft 1 is rotated by the electromagnetic force generated between the rotor 2 and the stator 3. That is, the rotor 2 and the stator 3 constitute a one rotor-two stator type axial gap motor.

[0017] The cylinder 4 (upper cylinder 4A and lower cylinder 4B) abuts on one side (upper side) of the upper stator 3A in the direction of the axis Ac and the other side (lower side) of the lower stator 3B in the direction of the axis Ac. Each of the upper cylinder 4A and the lower cylinder 4B has a cylindrical shape centered on the axis Ac. The above-mentioned upper eccentric shaft 11A and the ring-shaped rotary piston 12 (upper rotary piston 12A) fitted in the upper eccentric shaft 11A are accommodated inside the upper cylinder 4A. The above-mentioned lower eccentric shaft 11B and the ring-shaped rotary piston 12

(lower rotary piston 12B) fitted in the lower eccentric shaft 11B are accommodated inside the lower cylinder 4B. Further, intake ports 8A and 8B for guiding the refrigerant from the outside are provided in a portion of the upper cylinder 4A and the lower cylinder 4B in the circumferential direction, respectively.

[0018] The end plates 5 (upper end plate 5A, lower end plate 5B) abut on one side (upper side) of the upper cylinder 4A in the direction of the axis Ac and the other side (lower side) of the lower cylinder 4B in the direction of the axis Ac, respectively. That is, the upper end plate 5A sandwiches the upper cylinder 4A together with the back yoke 31A from the direction of the axis Ac. Similarly, the lower end plate 5B sandwiches the lower cylinder 4B together with the back yoke 31B from the direction of the axis Ac. Each of the upper end plate 5A and the lower end plate 5B has a disk shape centered on the axis Ac. The bearings 6 (upper bearing 6A, lower bearing 6B) are attached to a portion including the centers of the upper end plate 5A and the lower end plate 5B, respectively. A shaft end of the shaft main body 1H is supported by these bearings 6. Further, the upper end plate 5A and the lower end plate 5B are fixed to an inner peripheral surface of the housing 7 in a tightly fitted state.

[0019] As shown in an enlarged manner in Fig. 2, a surface of the upper end plate 5A facing the other side (lower side) in the direction of the axis Ac is an end plate main surface 5S. Further, a surface of the back yoke 31A facing one side (upper side) of the direction of the axis Ac is a back yoke facing surface 31S. An upper compression chamber C1 is formed by the end plate main surface 5S, the back yoke facing surface 31S, and the inner peripheral surface of the upper cylinder 4A. That is, in the present embodiment, a portion (back yoke 31A) of the stator 3 also serves as a member forming a portion of the upper compression chamber C1. Further, a lower compression chamber C2 is also formed by the back yoke 31B of the lower stator 3B, the lower end plate 5B, and the lower cylinder 4B, similarly to the upper compression chamber C1.

[0020] In the upper compression chamber C1 and the lower compression chamber C2, the above-mentioned upper rotary piston 12A and lower rotary piston 12B rotate eccentrically, respectively. As a result, volumes of the upper compression chamber C1 and the lower compression chamber C2 change with time, and the refrigerant taken in from the intake ports 8A and 8B is compressed. The compressed refrigerant passes through the inside of the housing 7 and is taken out from a discharge port 7A.

[0021] It is also possible to adopt a configuration in which the pressure of the refrigerant is increased in two stages by sequentially passing the upper compression chamber C1 and the lower compression chamber C2, and it is also possible to adopt a configuration in which the upper compression chamber C1 and the lower compression chamber C2 function independently.

(Action Effect)

[0022] According to the above configuration, the back yokes 31A and 31B of the stator 3 form the compression chambers C1 and C2 together with the end plate 5 and the cylinder 4. In other words, the back yokes 31A and 31B have both a function as a portion of the motor and a function as a portion of the members forming the compression chambers C1 and C2. As a result, the number of parts can be reduced. Further, the size of the device in the direction of the axis Ac can be suppressed by the reduced members.

[0023] Further, according to the above configuration, since the bearing 6 is provided in each of the pair of end plates 5, the shaft 1 can be supported by both end portions thereof. As a result, noise and vibration are reduced, and the shaft 1 can be rotated more stably.

[Second Embodiment]

[0024] Next, a second embodiment of the present disclosure will be described with reference to Fig. 3. The same components as those in the first embodiment are designated by the same reference numerals, and detailed description thereof will be omitted. As shown in Fig. 3, in the present embodiment, a bearing 6' (upper bearing 6A') is integrally provided in a back yoke 31A instead of an end plate 5' (upper end plate 5A'). As a result, the end plate 5' has a disk shape centered on an axis Ac, and no opening or the like is formed in the portion including the center. Further, although not shown in detail, another bearing 6' located at the lower portion is also integrally provided on a back yoke 31B like the upper bearing 6A'.

[0025] According to the above configuration, since the bearing 6' is provided in each of the pair of back yokes 31A and 31B, an end portion of a shaft 1 does not protrude from the end plate 5' side. That is, the dimension of the shaft 1 in the direction of the axis Ac can be kept small. As a result, the possibility that the shaft 1 is bent or misaligned can be further reduced. As a result, a rotary compressor 100 can be operated more stably.

(Other Embodiments)

[0026] The embodiments of the present disclosure have been described above. It is possible to make various changes and modifications to the above configuration as long as it does not deviate from the gist of the present disclosure. For example, as a modification example of the second embodiment, as shown in Fig. 4, the back yoke 31A can be configured to include an annular back yoke main body 34 integrally formed with the teeth 32A and a support plate 35 separately provided from the back yoke main body 34. The support plate 35 has a disk shape centered on the axis Ac, and a bearing 6' is provided at the center of the support plate 35. According to such a configuration, when the coil 33A is configured, the back yoke main body 34 is detachable from the

support plate 35, and thus, ease of manufacturing can be further improved.

[0027] Further, the configuration described in the first embodiment (the configuration in which the bearing 6 is provided in the end plate 5) and the configuration described in the second embodiment (the configuration in which the bearing 6' is provided in the back yokes 31A and 31B) can be combined. That is, it is possible to adopt a configuration in which the upper bearing 6A is attached to the end plate 5 and the lower bearing 6B is attached to the back yoke 31B, or a configuration in which the upper bearing 6A is attached to the back yoke 31A and the lower bearing 6B is attached to the end plate 5.

15 [Additional Notes]

[0028] The rotary compressor 100 described in each embodiment is grasped as follows, for example.

[0029]

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(1) A rotary compressor 100 according to a first aspect includes a shaft 1 that extends along an axis Ac, a disk-shaped rotor 2 that is fixed to the shaft 1 and centered on the axis Ac, a pair of stators 3 that faces the rotor 2 from both sides in a direction of the axis Ac, and includes disk-shaped back yokes 31A and 31B centered on the axis Ac, teeth 32A and 32B protruding from the back yokes 31A and 31B, and coils 33A and 33B wound around the teeth 32A and 32B, a pair of cylinders 4 that abuts on the stator 3 from the direction of the axis Ac and has an annular shape centered on the axis Ac, a pair of rotary pistons 12 that rotates eccentrically with the shaft 1, a pair of end plates 5 that forms compression chambers C1 and C2 for accommodating the rotary piston 12 together with the back yokes 31A and 31B by sandwiching the cylinder 4 together with the stator 3 from the direction of the axis Ac, and a pair of bearings 6 that is provided on at least one of the end plate 5 and the back yokes 31A and 31B.

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[0030] According to the above configuration, the back yokes 31A and 31B of the stator 3 form the compression chambers C1 and C2 together with the end plate 5 and the cylinder 4. In other words, the back yokes 31A and 31B have both a function as a portion of the motor and a function as a portion of the members forming the compression chambers C1 and C2. As a result, the number of parts can be reduced. Further, the size of the device in the direction of the axis Ac can be suppressed by the reduced members.

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[0031] (2) In the rotary compressor 100 according to a second aspect, the bearing 6 may be integrally provided in each of the pair of end plates 5.

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[0032] According to the above configuration, since the bearing 6 is provided in each of the pair of end plates 5, the shaft 1 can be supported by both end portions thereof. As a result, noise and vibration are reduced, and the shaft

1 can be rotated more stably.

[0033] (3) In the rotary compressor 100 according to a third aspect, the bearing 6 may be integrally provided in each of the pair of back yokes 31A and 31B.

[0034] According to the above configuration, since the bearing 6 is provided in each of the pair of back yokes 31A and 31B, the end portion of the shaft 1 does not protrude from the end plate 5 side. That is, the dimension of the shaft 1 can be kept small. This makes it possible to reduce the possibility that the shaft 1 is bent or misaligned.

[0035] (4) In the rotary compressor 100 according to a fourth aspect, the bearing 6 may protrude from the back yokes 31A and 31B in a direction toward the rotor 2.

[0036] According to the above configuration, since the bearing 6 protrudes from the back yokes 31A and 31B in the direction toward the rotor 2, it is possible to secure a large dimension of the bearing 6 in the direction of the axis Ac. This makes it possible to stably support the shaft 1 even when the load is high.

Industrial Applicability

[0037] The present disclosure relates to a rotary compressor.

[0038] According to the present disclosure, it is possible to provide a rotary compressor having a reduced number of parts and a smaller size.

Reference Signs List

[0039]

100 Rotary compressor
 1 Shaft
 1H Shaft main body
 11A Upper eccentric shaft
 11B Lower eccentric shaft
 12 Rotary piston
 12A Upper rotary piston
 12B Lower rotary piston
 2 Rotor
 21 Rotor core
 22 Permanent magnet
 3 Stator
 3A Upper stator
 3B Lower stator
 31A, 31B Back yoke
 31S Back yoke facing surface
 32A, 32B Teeth
 33A, 33B Coil
 4 Cylinder
 4A Upper cylinder
 4B Lower cylinder
 5 End plate
 5A Upper end plate
 5B Lower end plate
 5S End plate main surface
 6 Bearing

6A Upper bearing
 6B Lower bearing
 7 Housing
 7A Discharge port
 8A, 8B Intake port
 Ac Axis
 C1 Upper compression chamber
 C2 Lower compression chamber

Claims

1. A rotary compressor comprising:

a shaft (1) that extends along an axis (Ac);
 a disk-shaped rotor (2) that is fixed to the shaft (1) and centered on the axis (Ac);
 a pair of stators (3, 3A, 3B) that faces the rotor (2) from both sides in a direction of the axis (Ac), and includes each a disk-shaped back yoke (31A, 31B) centered on the axis (Ac), teeth (32A, 32B) protruding from the back yoke (31A, 31B), and a coil (33A, 33B) wound around the teeth (32A, 32B);
 a pair of cylinders (4) that abuts on the stator (3, 3A, 3B) from the direction of the axis (Ac) and has an annular shape centered on the axis (Ac);
 rotary pistons (12, 12A, 12B) that rotate eccentrically with the shaft(1);
 a pair of end plates (5, 5A, 5B) that forms respective compression chambers (C1, C2) for accommodating the rotary pistons (12, 12A, 12B);
 and a pair of bearings (6, 6A, 6B) that is provided on at least one of the end plate (5, 5A, 5B) and the back yoke (31A, 31B), **characterized in that** each compression chamber (C1, C2) is formed by one of the end plates (5, 5A, 5B) and one of the back yokes (31A, 31B) that is a portion of a stator (3, 3A, 3B), each cylinder (4, 4A, 4B) is sandwiched by the end plate (5, 5A, 5B) and the stator (3, 3A, 3B) from the direction of the axis (Ac).

2. The rotary compressor according to claim 1, wherein the bearing(6, 6A, 6B) is integrally provided in each of the pair of end plates(5, 5A, 5B).

3. The rotary compressor according to claim 1 or 2, wherein the bearing(6, 6A, 6B) is integrally provided in each of the pair of back yokes(31A, 31B).

4. The rotary compressor according to claim 3, wherein the bearing(6, 6A, 6B) protrudes from the back yoke(31A, 31B) in a direction toward the rotor(2).

Patentansprüche

1. Rotationsverdichter, umfassend:

eine Welle (1), die sich entlang einer Achse (Ac) 5
erstreckt;
einen scheibenförmigen Rotor (2), der an der
Welle (1) befestigt und auf der Achse (Ac) zentriert ist;
ein Paar von Statoren (3, 3A, 3B), das dem Rotor 10
(2) von beiden Seiten in einer Richtung der
Achse (Ac) zugewandt ist und jeweils ein scheibenförmiges hinteres Joch (31A, 31B), das auf der Achse (Ac) zentriert ist, Zähne (32A, 32B), die von dem hinteren Joch (31A, 31B) vorstehen, und eine Spule (33A, 33B), die um die Zähne (32A, 32B) gewickelt ist, einschließt;
ein Paar von Zylindern (4), das aus der Richtung der Achse (Ac) auf den Stator (3, 3A, 3B) angrenzt und eine ringförmige Form aufweist, die auf der Achse (Ac) zentriert ist;
Rotationskolben (12, 12A, 12B), die sich exzentrisch mit der Welle (1) drehen;
ein Paar von Endplatten (5, 5A, 5B), die jeweilige Verdichtungskammern (C1, C2) zur Aufnahme der Rotationskolben (12, 12A, 12B) bilden;
und ein Paar von Lagern (6, 6A, 6B), das auf mindestens einer der Endplatten (5, 5A, 5B) und dem hinteren Joch (31A, 31B) bereitgestellt ist, **dadurch gekennzeichnet, dass** jede Verdichtungskammer (C1, C2) durch eine der Endplatten (5, 5A, 5B) und eines der hinteren Joche (31A, 31B), das ein Abschnitt eines Stators (3, 3A, 3B) ist, gebildet ist, wobei jeder Zylinder (4, 4A, 4B) durch die Endplatte (5, 5A, 5B) und den Stator (3, 3A, 3B) aus der Richtung der Achse (Ac) sandwichartig eingeschlossen ist.

2. Rotationsverdichter nach Anspruch 1, wobei das Lager (6, 6A, 6B) in jedem des Paares von Endplatten (5, 5A, 5B) integral bereitgestellt ist. 40
3. Rotationsverdichter nach Anspruch 1 oder 2, wobei das Lager (6, 6A, 6B) in jedem des Paares von hinteren Jochen (31A, 31B) integral bereitgestellt ist. 45
4. Rotationsverdichter nach Anspruch 3, wobei das Lager (6, 6A, 6B) von dem hinteren Joch (31A, 31B) in einer Richtung zu dem Rotor (2) hin vorsteht. 50

Revendications

1. Compresseur rotatif comprenant :

un arbre (1) qui s'étend le long d'un axe (Ac) ;
un rotor en forme de disque (2) fixé à l'arbre (1) et centré sur l'axe (Ac) ;

une paire de stators (3, 3A, 3B) qui fait face au rotor (2) des deux côtés dans la direction de l'axe (Ac), et inclut chacun une fourche arrière en forme de disque (31A, 31B) centrée sur l'axe (Ac), des dents (32A, 32B) faisant saillie de la fourche arrière (31A, 31B), et une bobine (33A, 33B) enroulée autour des dents (32A, 32B) ;
une paire de cylindres (4) qui s'appuie sur le stator (3, 3A, 3B) dans la direction de l'axe (Ac) et qui a une forme annulaire centrée sur l'axe (Ac) ;
des pistons rotatifs (12, 12A, 12B) qui tournent excentriquement avec l'arbre (1) ;
une paire de plaques d'extrémité (5, 5A, 5B) qui forme des chambres de compression respectives (C1, C2) pour accueillir les pistons rotatifs (12, 12A, 12B) ;
et une paire de roulements (6, 6A, 6B) qui est fournie sur au moins une des plaques d'extrémité (5, 5A, 5B) et l'arcade arrière (31A, 31B), **caractérisé en ce que** chaque chambre de compression (C1, C2) est formée par l'une des plaques d'extrémité (5, 5A, 5B) et l'une des culasses (31A, 31B) qui est une partie d'un stator (3, 3A, 3B), chaque cylindre (4, 4A, 4B) est pris en sandwich par le plateau d'extrémité (5, 5A, 5B) et le stator (3, 3A, 3B) dans la direction de l'axe (Ac).

2. Compresseur rotatif selon la revendication 1, dans lequel le palier (6, 6A, 6B) est intégré dans chacune des paires de plaques d'extrémité (5, 5A, 5B).
3. Compresseur rotatif selon la revendication 1 ou 2, dans lequel le palier (6, 6A, 6B) est intégré dans chacune des paires d'arcades arrière (31A, 31B).
4. Compresseur rotatif selon la revendication 3, dans lequel le palier (6, 6A, 6B) fait saillie de l'arcade arrière (31A, 31B) en direction du rotor (2).

FIG. 2

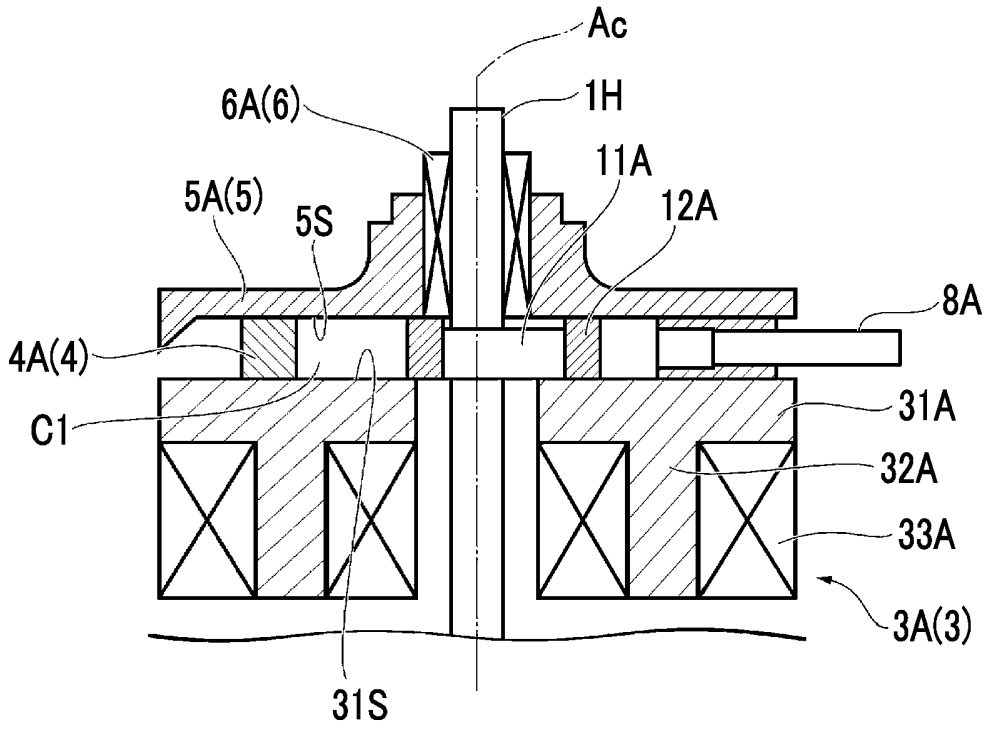


FIG. 3

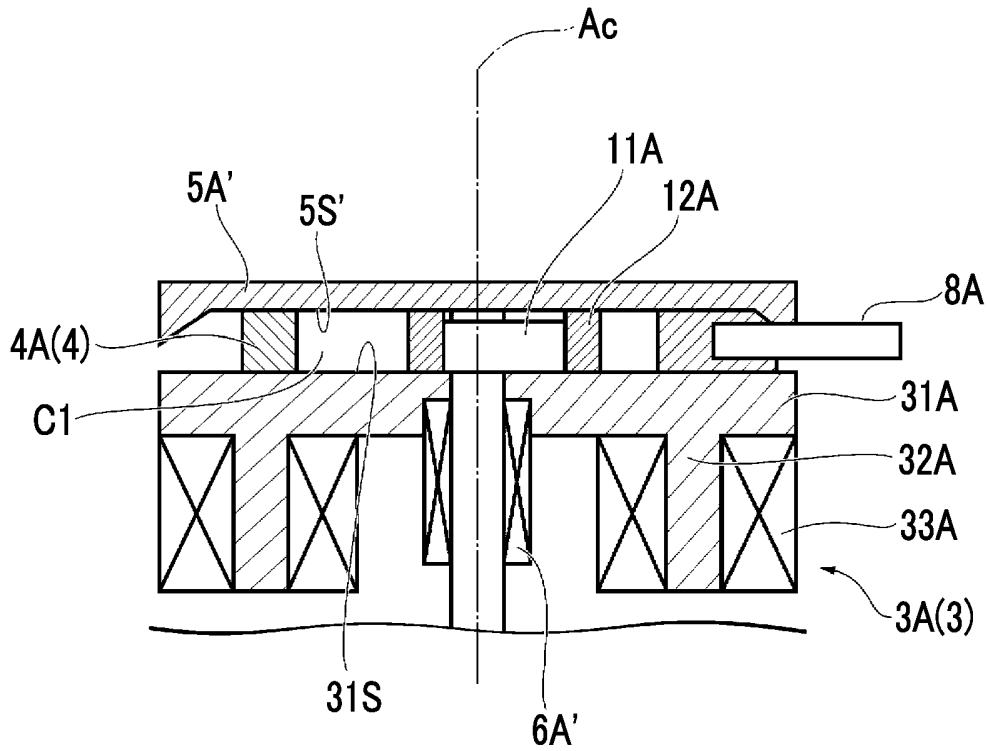
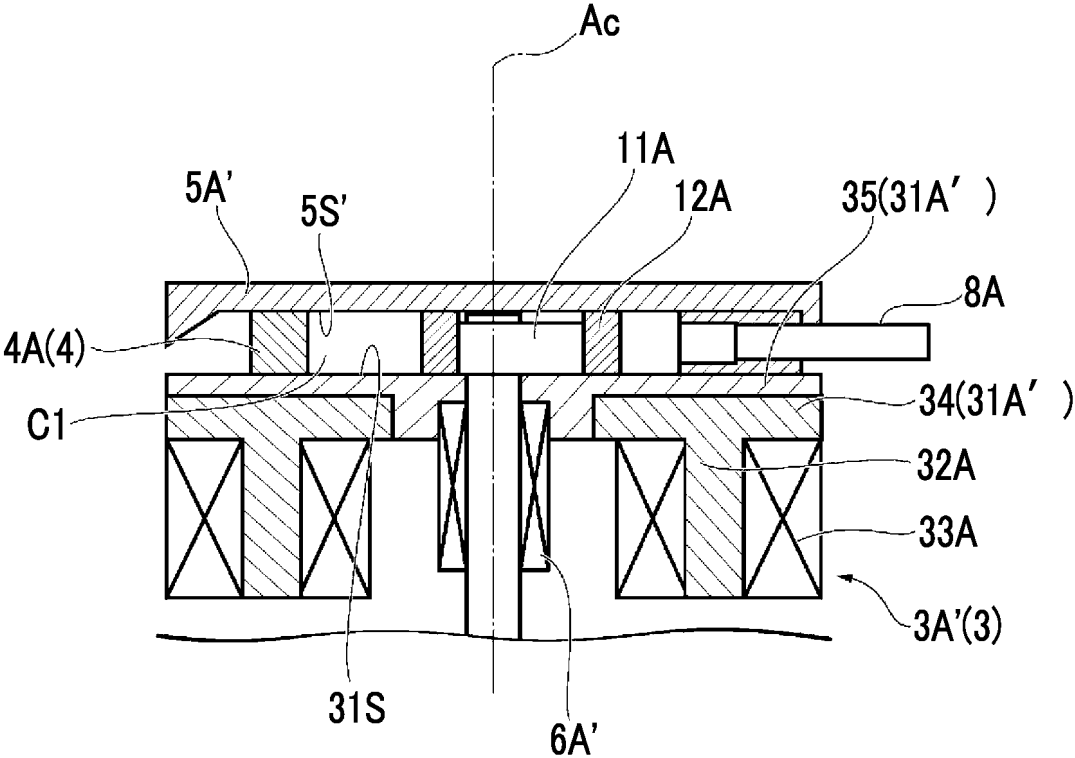


FIG. 4



REFERENCES CITED IN THE DESCRIPTION

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