

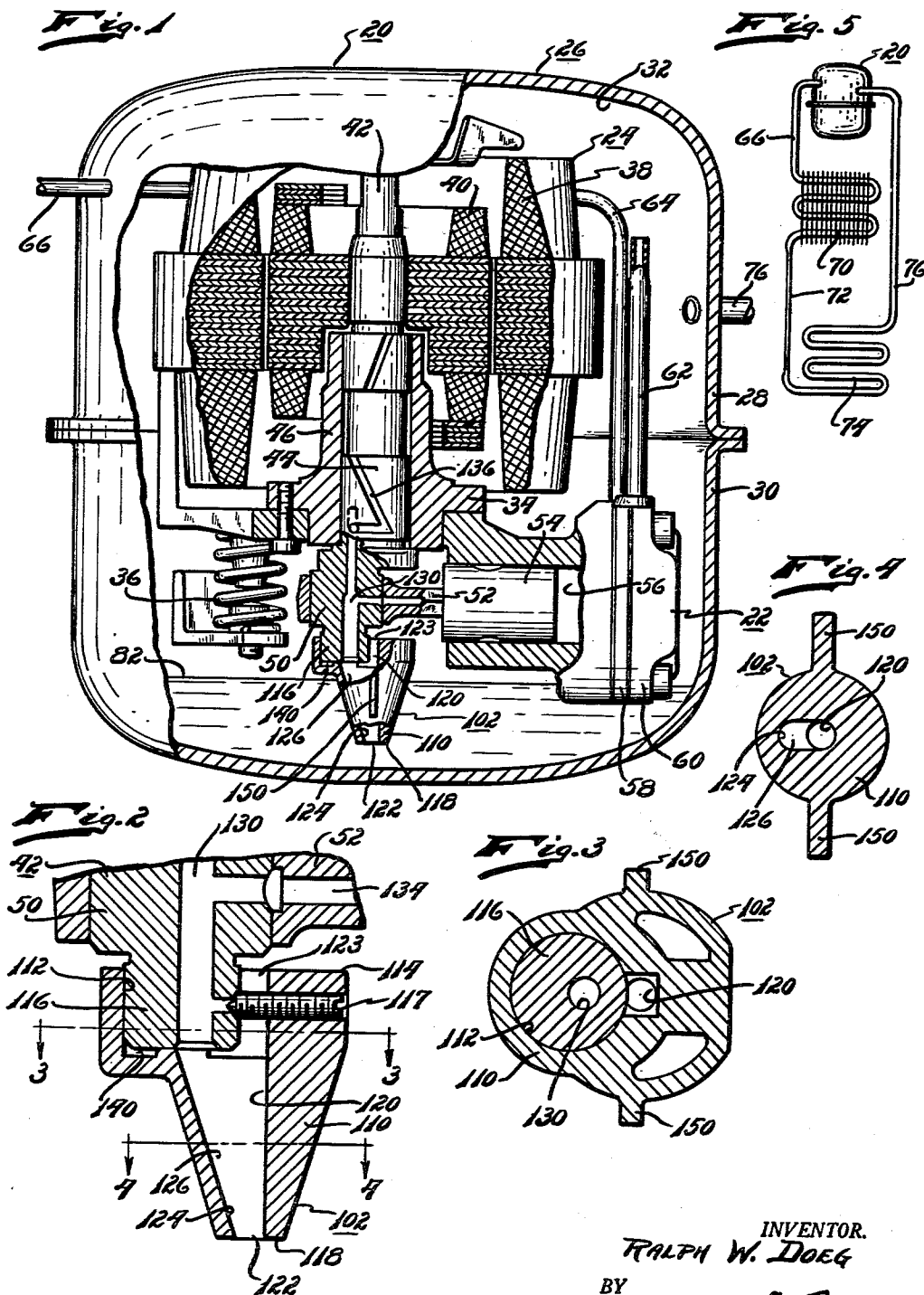
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REFRIGERATING APPARATUS

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REFRIGERATING APPARATUS

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My invention relates to a refrigerating apparatus and more particularly to hermetically sealed motor compressor units for use in connection with such apparatus.

It is an object of the present invention to provide an improved arrangement for lifting and circulating lubricant from a reservoir to the motor compressor unit.

Another object of the present invention is to provide an improved arrangement for lifting and circulating lubricant from a reservoir to a motor compressor unit, to separate any refrigerant mixed with the lubricant and to divert such separated refrigerant from entering the lubricant conducting passages in the motor compressor unit with the lubricant.

Another object of the present invention is to provide an improved arrangement for lifting and circulating lubricant from a reservoir to a motor compressor unit, to separate any foreign bodies from the lubricant and to retain such foreign bodies to prevent their entry with the lubricant into the lubricant conducting passages in the motor compressor unit.

Another object of the present invention is to provide an improved oil or lubricant lifting device which is simple in construction, economical of manufacture and highly efficient in use.

Further objects and advantages of the present invention will be apparent from the following description, reference being had to the accompanying drawing, wherein a preferred form of the present invention is clearly shown.

FIG. 1 is a front view of a refrigerant compressor embodying features of my invention and showing a portion thereof broken away;

FIG. 2 is a vertical fragmentary view of my invention;

FIG. 3 is a view taken along line 3—3 of FIG. 2;

FIG. 4 is a view taken along line 4—4 of FIG. 2; and

FIG. 5 is a diagrammatic view of a refrigerating system including a compressor embodying features of my invention.

Shown in the drawing is a motor-compressor unit 20 which comprises in general a compressor 22 and an electric motor 24 mounted within a sealed casing or shell 26.

The casing 26 consists of an upper shell 28 and a lower shell 30 which are welded together by their flanges to form a sealed chamber 32.

The motor 24 and the compressor 22 are secured to a common casting 34 to form an assembled unit which is mounted by suitable springs 36 to the inner wall of the casing 26 in a vertically extending position within the sealed chamber 32.

The motor 24 includes a stator 38 that is secured on the casting 34 and a rotor 40 which is mounted upon a crankshaft 42.

The crankshaft 42 has a shaft section 44 that is journaled in a bearing section 46 formed in the casting 34. The shaft section 44 extends upwardly and outwardly from the bearing 46 and supports the rotor 40. Adjacent the lower end of the bearing 46 the shaft is formed with a crankarm 50 to which is connected one end of a connecting rod 52. The opposite end of the connecting rod 52 connects to a piston 54 which operates in a cylinder 56. The outer end of the cylinder 56 is closed by a valve plate 58 and a cylinder head 60.

The valve plate 58 is provided with a suitable inlet valve (not shown) to admit refrigerant vapors from the chamber 32 into the cylinder 56. The refrigerant is con-

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ducted from the chamber 32 toward the cylinder 56 through a conduit 62. The valve plate is also provided with suitable discharge valves (not shown) to control the flow of compressed refrigerant vapors from the cylinder into the cylinder head. From the cylinder head the compressed refrigerant flows through a conduit 64 and through a connecting conduit 66, external of the casing 26, into a condenser 70 for cooling and condensing. The condensed refrigerant is metered in its flow from the condenser 70 by a small diameter conduit 72 for conduction into the evaporator 74. From the evaporator 74 the heat laden refrigerant is returned to the chamber 32 through a conduit 76.

A reservoir of oil or lubricant is maintained in the bottom of the chamber 32. As oil is a conductor of sound it is highly desirable that the oil be maintained at a level 82 which is preferably below the compressor body. With the oil level at 82 there is a minimum of contact with the compressor body which reduces to a minimum the conduction of operational noises of the compressor through the oil to the outer casing. To provide lubricant for the compressor from the spatially distanced reservoir I have provided a lubricant lift or oil lifting device 102.

During the off-phase of the refrigerating cycle the conduit 72 permits equalization of pressures in the casing 26 and the condenser 70. When the pressures equalize there will be an increase in pressure in the chamber 32 and this results in the refrigerant becoming absorbed in the lubricant. During the on-phase of the compressor the refrigerant will leave the lubricant due to the reduction of pressure in the chamber 32. This causes foaming and the result is that the lubricant lift 102 separates the lubricant from the refrigerant.

The oil lifting device 102 comprises an inverted cone shaped body 110 which is secured upon for carry by the crankarm 50. The cone shaped body 110 is formed with a bore 112 extending inwardly from the base side 114 to receive a downwardly projecting crankarm member 116. The cone is secured to the crankarm member 116 by a locking screw 117. The bore 112 is off-centered in the cone from the cone's axis, a distance equal to the radius from the center of the crankarm to the axis of the crankshaft so that when the cone is mounted on the crankarm the axis of the cone is in direct alignment with the axis of the shaft section 44 of the crankshaft. The axis of the cone is an imaginary centerline from the truncated apex 118 extending perpendicularly to the base side 114.

Extending through the cone body 110 along the axis of the cone is a central passageway 120 which has its inlet opening 122 in the apex 118 and its outlet 123 in the base side 114. The passage 120, from the entrance opening 122, has a side wall that inclines, as at 124, towards the bore 112 to form an inclined or angular passage 126 which communicates with a longitudinally directed passage 130 formed in the crankshaft 42.

In operation as the crankarm rotates in an orbital path it will carry the cone therearound, but as the apex 118 of the cone is aligned with the axis of the shaft 44 it will rotate on a stationary axis and minimize any disturbance to the oil to permit its entrance into the opening 122. The lower end of the cone body extends below the oil level 82 so that at all times the inlet opening 122 is submerged in the oil. The outlet 123 of passage 120 is above the oil level 82. As the cone rotates, the oil entering the opening 122 is centrifugally lifted up the inclined side 124, or passage 126, to enter the passage 130 of the crankshaft from which the oil is distributed to a radial passage to lubricate the crankarm to connecting rod connection, and for conduction through a passageway 134 formed in the connecting rod for flow to the piston for lubricating the cylinder walls and also the wrist pin connection (not shown). The passage 130 also connects to a radial pas-

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sage for conducting lubricant to the circumferential grooves 136 formed on the shaft section 44.

As the lubricant or oil is subjected to centrifugal force to effect its lifting within the cone's body this force will also cause any lubricant mixed with the refrigerant to become separated therefrom, and after separation the refrigerant will travel vertically upwardly through the axial passage 120 to the outlet 123 for return to the chamber 32.

A circumferential recess or pocket 140 is formed in the bottom wall of the bore 112 which is in open communication with the inclined passageway 126. Dirt and other foreign particles that are in the oil will be centrifugally separated therefrom to become trapped and remain lodged in the recess 140. The inclined side 124 is so arranged that the dirt particles will be centrifuged to pass into the recess 140 before the oil can enter the passage 130.

Though the oil level 82 is below the compressor body there still remains direct contact by the oil with conduits, mufflers and some projecting parts of the compressor through which operational sounds are carried and conducted to the oil for transmittal to the outer casing. To reduce this transmittal of objectionable sounds the cone body 110 is provided with a pair of paddles 150 which extend radially outwardly from the axis of the cone in diametrically opposed relation. These paddles are more fully described and are the subject of a co-pending application for U.S. Letters Patent, Serial No. 841,547, filed by Nicholas J. Bohn on even date herewith, now Patent No. 2,990,111, and assigned to the same assignee as this application.

The paddles 150 are preferably formed integral with the cone's body and terminate at their lower end a spatial distance above the apex 118. The paddles serve to agitate the oil so as to mix into the oil refrigerant vapors from the chamber 32. The refrigerant vapors are so mixed and dispersed into the oil as to form small bubbles throughout the entire body of oil maintained in the reservoir to reduce the conductivity of sound through the oil to the outer casing. By terminating the paddles a spatial distance from the apex 118 the oil is not disturbed by the paddles to interfere with its entrance into the inlet opening 122.

Although only a preferred form of the invention has been illustrated and that form in detail it will be apparent to those skilled in the art that various modifications may be made therein without departing from the spirit of the invention or from the scope of the appended claims.

I claim:

1. Refrigerating apparatus comprising a casing forming a sealed chamber, a body of lubricant stored in the lower portion of said chamber, a compressor including a vertically extending crankshaft within said chamber, said crankshaft terminating a spatial distance from the lubricant, a motor within said chamber operatively connected to drive said crankshaft, a lubricant conducting means secured to said crankshaft for conveying lubricant

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stored in said chamber to said crankshaft including a body having a passage with an inlet below the level of lubricant and an outlet above the level of the lubricant, said body being secured to said crankshaft for rotation therewith with said passage extending in axial alignment with the axis of said crankshaft, said passage being inclined to diverge from the opposing side thereof as it extends from said inlet opening, said crankshaft having a passage and an opening in said terminated end thereof, and said inclined side terminating in alignment with said opening in said crankshaft to direct thereinto the flow of lubricant from said inclined side, a recess formed in said body, and said recess being extended about within said body and terminating in opposed and open communication with said passage whereby said recess is adapted to receive and trap foreign matter centrifuged from the liquid lubricant as it is conveyed through said body prior to said lubricant entering said crankshaft.

2. Refrigerating apparatus comprising a casing, a sealed chamber formed by said casing, a body of liquid lubricant stored in the lower portion of said chamber, a compressor including a crankshaft within said chamber, said crankshaft extending in a vertical position within said chamber, said crankshaft having a crankarm at its lower end, a projecting member on said crankarm extending downwardly therefrom and terminating a spatial distance from the liquid lubricant, a motor within said chamber operatively connected to drive said crankshaft, a lubricant conducting means for conveying lubricant stored in said chamber to said crankshaft including a body secured to said projecting member, said body having an inwardly extended bore to receive said projecting member, a vertical passage extending through said body having its inlet below said level of liquid lubricant and its outlet above said level of liquid lubricant, said vertical passage being formed in said body to extend axially in alignment with the axis of rotation of said crankshaft, a side of said vertical passage being inclined from adjacent said inlet opening to and into said bore to form an inclined passage from said opening into said bore, a passageway in said crankshaft extending to an opening in said projecting member, said opening in said projecting member being in open communication with said inclined passage, a horizontally extended recess formed in and extending about said body, and said recess being extended to terminate in opposed and in open communication with said inclined passage to receive foreign matter being centrifuged from the lubricant as it is lifted by centrifugal force up said inclined passage.

References Cited in the file of this patent

UNITED STATES PATENTS

1,807,830	Cattaneo	June 2, 1931
1,862,724	Summers	June 14, 1932
2,287,203	Smith	June 23, 1942
2,440,812	Simpson	May 4, 1948
2,898,072	Buschmann	Aug. 4, 1959