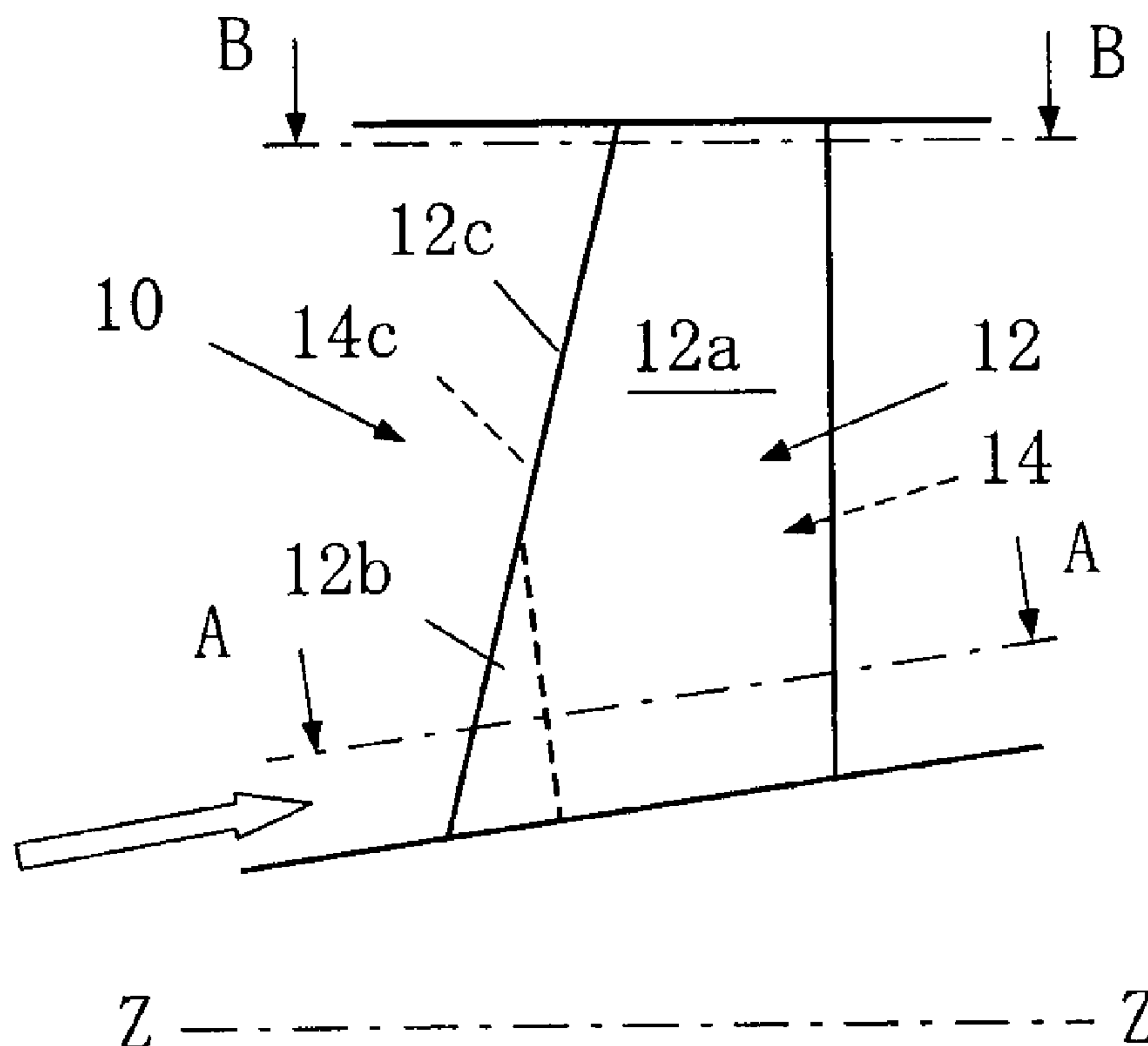




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(54) Title: BLADE ROW OF AXIAL FLOW TYPE COMPRESSOR



(57) Abrégé/Abstract:

In a blade row of an axial flow type compressor in which a rotor blade row and a stator blade row are alternately arranged in an axial direction, the stator blade row (10) is formed by plural main stator blades (12) and plural sub-stator blades (14). Each main stator blade (12) is formed by a basic blade portion (12a) which has the same shape as that of each sub-stator blade and a forward blade portion (12b) which extends to the upstream side of the basic blade portion. The basic blade portion (12a) is located at the same position in an axial direction. The forward blade portion (12b) forms a forward stator blade row which has a circumferential interval larger than that of the basic stator blade row in the vicinity of at least a radial inner end.



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ABSTRACT

In a blade row of an axial flow type compressor in which a rotor blade row and a stator blade row are alternately arranged in an axial direction, the stator blade row (10) is formed by plural main stator blades (12) and plural sub-stator blades (14). Each main stator blade (12) is formed by a basic blade portion (12a) which has the same shape as that of each sub-stator blade and a forward blade portion (12b) which extends to the upstream side of the basic blade portion. The basic blade portion (12a) is located at the same position in an axial direction. The forward blade portion (12b) forms a forward stator blade row which has a circumferential interval larger than that of the basic stator blade row in the vicinity of at least a radial inner end.

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DESCRIPTION

BLADE ROW OF AXIAL FLOW TYPE COMPRESSOR

BACKGROUND OF THE INVENTION5 Technical Field of the Invention

[0001]

The present invention relates to a blade row of an axial flow type compressor in which a rotor blade row and a stator blade row are alternately arranged in an axial
10 direction.

Description of the Related Art

[0002]

In a gas turbine or a jet engine, a compressor for
15 compressing an air introduced from the outside is configured as an axial flow type compressor in which a rotor blade row and a stator blade row are arranged in an axial direction.

[0003]

20 In the axial flow type compressor, since an inflow mach number becomes high at a position on the side of a radial inner diameter (on the hub side) of a stator blade forming the stator blade row under the condition of a high flow rate and a high pressure, choking easily occurs in a
25 minimum valid passageway sectional portion (throat area), thereby increasing pressure loss. Additionally, the flow rate cannot increase any more when the choking occurs.

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[0004]

In the axial flow type compressor, a chord length may be increased in order to realize a high pressure at a position on the side of the radial inner diameter (on the hub side) of a rotor blade forming the rotor blade row. However, since friction loss also increases, the advantage of the increased chord length becomes small. Since a relative inflow mach number is large at a position on the side of a radial outer diameter (on the tip side), pressure loss increases due to an acceleration before a throat area. Additionally, since the choking easily occurs, the flow rate cannot increase.

[0005]

Japanese Patent Application Laid-Open No. H06-257597 discloses a technique for reducing the above-described problems.

15 [0006]

A blade row structure of an axial flow type compressor is disclosed in this application to realize high flow rate and high efficiency of the compressor. As shown in Fig. 1, a blade row structure of an axial flow type compressor includes a series of blades arranged between an outer passageway wall and an inner passageway wall and in an annular shape to have an interval therebetween in a circumferential direction. The inner passageway wall is provided with a concave portion which is located at a throat portion, in which a passageway sectional area in the row of the blades becomes minimum, so as to expand a passageway

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sectional area and to provide a smooth convex portion located on the downstream side of the concave portion. This arrangement suppresses a deceleration of a fluid flowing through a base portion on the rear side of the blade.

5 [0007]

Additionally, US Patent No. 5,002,461 and US Patent No. 5,639,217 disclose a centrifugal compressor different from the axial flow type compressor.

[0008]

10 US Patent No. 5,002,461, as shown in Fig. 2 discloses an impeller including a hub, a plurality of main blades formed in the hub, and a plurality of splitter blades formed in the hub. In this impeller, each splitter blade is formed between the adjacent main blades.

15 [0009]

US Patent No. 5,639,217, as shown in Fig. 3, discloses an impeller including a rotary disc which has a hub suitable for a rotary shaft. A plurality of full blades are formed on a surface of the rotary disc, and a plurality of splitter blades are formed
20 on the surface of the rotary disc. In this impeller, the full blades and the splitter blades are alternately arranged in a rotary direction of the rotary disc.

[0010]

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[0011]

As described above, in the axial flow type compressor, a problem arises in that pressure loss of the rotor blade row and the stator blade row increases in the case of a high inflow mach number, and a problem arises in that a choking occurs in the throat portion in the blade row and an inflow air flow rate is limited. In Japanese Patent Application Laid-Open No. H06-257597 described above, it is expected that a local advantage is exhibited, but a three-dimensional advantage is small.

10 Additionally, especially in the case of a fan, it is configured such that the number of the stator blades is larger than the number of rotor blades and a cutoff condition is produced that has advantages in noise characteristics. However, as described in Japanese Patent Application Laid-Open No. H06-
15 257597, in order to handle the high-mach-number fluid, it is necessary to expand an area between blades.

As expanding means, means for decreasing the number of stator blades may be considered. However, since the number of rotor blades is approximately equal to that of the stator blades,
20 a problem arises in that noise increases.

SUMMARY OF THE INVENTION

[0012]

The present invention is contrived to solve the above-described problems. That is, an object of the invention is to provide a blade row of an axial flow type compressor capable of more reducing pressure loss and of more improving an air flow rate than those of the conventional art in the case of a high inflow mach number by three-dimensionally and actively adjusting a blade shape.

[0013]

According to the invention, there is provided a blade row of an axial flow type compressor in which a rotor blade row and a stator blade row are alternately arranged in an axial direction, wherein the stator blade row is formed by plural main stator blades which are located in a circumferential direction of a rotary axis of the rotor blade row so as to have an interval therebetween and plural sub-stator blades which are located between the main stator blades in a circumferential direction so as to have an interval therebetween, wherein each main stator blade is formed by a basic blade portion which has the same shape as that of each sub-stator blade and a forward blade portion which extends to the upstream side of the basic blade portion, wherein the basic blade portion of the main stator blade and the sub-stator blade are located at the same position in an axial direction so as to form a basic stator blade row therebetween, and wherein the forward blade

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portion of the main stator blade forms a forward stator blade row which has a circumferential interval larger than that of the basic stator blade row in the vicinity of at least a radial inner end.

5 [0014]

According to the invention, there is provided a blade row of an axial flow type compressor in which a rotor blade row and a stator blade row are alternately arranged in an axial direction, wherein the rotor blade row is
10 formed by plural main rotor blades which are located in a circumferential direction of a rotary axis thereof so as to have an interval therebetween and plural sub-rotor blades which are located between the main rotor blades in a circumferential direction so as to have an interval
15 therebetween, wherein each main rotor blade is formed by a basic blade portion which has the same shape as that of each sub-rotor blade and a forward blade portion which extends to the upstream side of the basic blade portion, wherein the basic blade portion of the main rotor blade and
20 the sub-rotor blade are located at the same position in an axial direction so as to form a basic rotor blade row therebetween, and wherein the forward blade portion of the main rotor blade forms a forward rotor blade row which has a circumferential interval larger than that of the basic
25 rotor blade row in the vicinity of at least a radial inner end.

[0015]

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According to the preferred embodiment of the invention, a front edge of the main rotor blade is located on the downstream side of a front edge of the sub-rotor blade from a radial middle portion to an outer end.

5 [0016]

According to the configuration of the invention, the stator blade row is formed by the basic stator blade row which is formed by the basic blade portion of the main stator blade and the sub-stator blade and the forward
10 stator blade row which is formed by only the forward blade portion of the main stator blade. The circumferential interval of the forward stator blade row is larger than that of the basic stator blade row (by approximately two times) in the vicinity of at least the radial inner end.
15 Accordingly, even in the case where a high-mach-number fluid flows into the stator blade row on the hub side, it is possible to expect a wide dynamic range, high efficiency, and an expansion of a throat area on the hub side determined by the interval of the forward blade row.

20 [0017]

Since the basic blade portion of the main stator blade has the same shape as that of the sub-stator blade from the vicinity of a mid-span except for the vicinity of the radial inner end to the tip side, the basic stator
25 blade row formed by the basic blade portion of the main stator blade and the sub-stator blade has the same configuration as that of the conventional stator blade row,

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and the number of rotor blades and stator blades is the same as that of the conventional art, thereby maintaining a cutoff condition which is advantageous in noise caused by the interference between the rotor blade and the stator
5 blade.

In addition, it is possible to realize a decrease in weight as a whole as much as the short sub-stator blade on the hub side.

[0018]

10 According to the above-described configuration of the invention, the rotor blade row is formed by the basic rotor blade row which is formed by the basic blade portion of the main rotor blade and the sub-rotor blade and the forward rotor blade row which is formed by only the forward
15 blade portion of the main rotor blade. The number of blades of the forward rotor blade row is smaller than that of (is a half of) the basic rotor blade row. Accordingly, it is possible to reduce the fluid friction loss of the blade portion and to efficiently increase the pressure.

20 [0019]

Since the circumferential interval of the forward rotor blade row in the vicinity of the radial inner end is larger than that of the basic rotor blade row (by approximately two times), it is possible to expect a wide
25 dynamic range, high efficiency, and an expansion of a throat area on the hub side determined by the interval of the forward blade row.

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[0020]

With the configuration in which the front edge of the main rotor blade is located on the downstream side of the front edge of the sub-rotor blade from the radial middle portion to the outer end, the circumferential interval of the front edge of the sub-rotor blade on the tip side is large (by approximately two times). Accordingly, it is possible to expand the throat area at the tip side and to expect the pressure loss reduction at a high-ratio flow rate.

In addition, it is possible to realize a decrease in weight as a whole as much as the short sub-rotor blade on the hub side.

[0021]

Accordingly, in any case of the stator blade row and the rotor blade row, it is possible to reduce pressure loss of the compressor, and to more increase an air flow rate while maintaining a compression characteristic than that of the conventional art.

[0022]

Further, the above-described advantage according to the invention is verified by means of the CFD (computer fluid dynamics) analysis.

BRIEF DESCRIPTION OF THE DRAWINGS

[0023]

Fig. 1 is a schematic view showing a blade row

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structure of an axial flow type compressor disclosed in Patent Document 1.

Fig. 2 is a schematic view showing Patent Document 2.

Fig. 3 is a schematic view showing Patent Document 3.

5 Fig. 4A is a view showing a blade row of an axial flow type compressor according to a first embodiment of the invention.

Fig. 4B is a view showing a blade row of an axial flow type compressor according to a second embodiment of the invention.

Fig. 4C is a sectional view taken along the line A-A of Figs. 4A and 4B.

Fig. 4D is a sectional view taken along the line B-B of Figs. 4A and 4B.

15 Fig. 5 is a diagrammatic view showing predicted performances according to the first and second embodiments.

Fig. 6 is a view showing CFD analysis results according to the first and second embodiments.

Fig. 7A is a view showing the blade row of the axial flow type compressor according to a third embodiment of the invention.

Fig. 7B is a sectional view taken along the line A-A of Fig. 7A.

Fig. 7C is a sectional view taken along the line B-B of Fig. 7A.

25 Fig. 8A is a view showing the blade row of the axial flow type compressor according to a fourth embodiment of

- 11 -

the invention.

Fig. 8B is a sectional view taken along the line A-A of Fig. 8A.

Fig. 8C is a sectional view taken along the line B-B
5 of Fig. 8A.

DESCRIPTION OF THE PREFERRED EMBODIMENT

[0024]

Hereinafter, preferred embodiments of the invention
10 will be described with reference to the accompanying drawings. Additionally, in the respective drawings, the same reference numerals are given to the same components, and the repetitive description thereof will be omitted.

Figs. 4A to 4C are examples in which the blade row
15 according to the invention is applied to a stator blade row. In these drawings, Fig. 4A shows a first embodiment, Fig. 4B shows a second embodiment, Fig. 4C is a sectional view taken along the line A-A, and Fig. 4D is a sectional view taken along the line B-B.

20 [0025]

Fig. 4A is a schematic side view showing a stator
blade row 10 according to the first embodiment of the
invention. In this drawing, the stator blade row 10
according to the invention is formed by plural main stator
25 blades 12 and plural sub-stator blades 14. In this drawing,
each sub-stator blade 14 is located on the rear side of
each main stator blade 12.

- 12 -

The plural main stator blades 12 are located in a circumferential direction of a rotary axis Z-Z of a rotor blade row (not shown) so as to have an interval therebetween. Additionally, the plural sub-stator blades 14 are located between the main stator blades 12 in a circumferential direction so as to have an interval therebetween. Accordingly, the number of the main stator blades 12 is the same as that of the sub-stator blades 14.

[0026]

The main stator blade 12 is formed by a basic blade portion 12a which has the same shape as that of the sub-stator blade 14 and a forward blade portion 12b which extends to the upstream side of the basic blade portion. Accordingly, the basic blade portion 12a of the main stator blade has the same configuration as that of the sub stator blade 14 except for the existence of the forward blade portion 12b.

[0027]

The basic blade portion 12a of the main stator blade 12 and the sub-stator blade 14 are located at the same position in an axial direction, and a basic stator blade row is formed therebetween. In this basic stator blade row, it is desirable to have a uniform circumferential interval between the basic blade portion 12a and the sub-stator blade 14, but the interval may be adjusted in accordance with a flow state.

[0028]

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The forward blade portion 12b of the main stator blade 12 forms a forward stator blade row which has a circumferential interval larger than that of the basic stator blade row 12a in the vicinity of at least a radial inner end (on a hub side). The circumferential interval of the forward stator blade row is approximately two times that of the basic stator blade row.

[0029]

Fig. 4B is a schematic side view showing the stator blade row 10 according to the second embodiment of the invention.

In this example, a front edge 12c of the main stator blade 12 is located on the upstream side of a front edge 14c of the stator blade 14 from a radial middle portion to an outer end.

The other configurations are the same as those of the first embodiment.

[0030]

According to the above-described configuration, as shown in Fig. 4C, it is possible to allow the circumferential interval of the forward stator blade row which is formed by the forward blade portions 12b to be larger than that of the basic stator blade row, which is formed by the basic blade portions 12a of the main stator blades 12 and the sub-stator blades 14, in the vicinity of at least the radial inner end (on the hub side) (by approximately two times). Accordingly, even in the case

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where a high-mach-number fluid 1 flows into the stator blade row on the hub side, it is possible to expect a wide dynamic range, high efficiency, and an expansion of a throat area 2 on the hub side determined by the interval of the forward blade row 12b.

[0031]

As shown in Fig. 4D, since the basic blade portion 12a of the main stator blade has the same shape as that of the sub-stator blade 14 from the vicinity of a mid-span except for the vicinity of the radial inner end to the tip side, the basic stator blade row formed by the basic blade portion 12a of the main stator blade 12 and the sub-stator blade 14 has the same configuration as that of the conventional stator blade row, and the number of rotor blades and stator blades is the same as that of the conventional art, thereby maintaining a cutoff condition which is advantageous in noise caused by the interference between the rotor blade and the stator blade.

In addition, it is possible to realize a decrease in weight as a whole as much as the short sub-stator blade 14 on the hub side.

[0032]

Fig. 5 is a diagrammatic view showing predicted performances according to the first and second embodiments. In this drawing, a lateral axis indicates a stator blade incident angle, and a longitudinal axis indicates a pressure loss coefficient. In the drawing, a broken line

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indicates a conventional stator blade row, and a solid line indicates a stator blade row according to the invention.

[0033]

As shown in this drawing, since the stator blade incident angle deviates from an optimal point when the flow rate increases or decreases with respect to a design point, the pressure loss coefficient largely increases. However, in the stator blade row according to the invention, since the number of blades of the forward stator blade row is smaller than that of (is a half of) the basic rotor blade row, even in the case where the fluid friction loss of the blade portion decreases and the stator blade incident angle varies, it is possible to reduce the pressure loss coefficient in a broad range and to efficiently increase the pressure.

[0034]

Fig. 6 is a comparative view showing streamlines of the blade surfaces according to the conventional art and the invention. In this drawing, "a base type" on the left side shows the streamline according to the conventional art, and "an invented type" on the right side shows the streamline according to the invention.

This drawing shows the streamline in the vicinity of a negative pressure surface in the state where a fluid flows from the right side to the left side of the blade. At a position on the downstream side (the right side of the drawing) surrounded by a circle, when a dark area

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(low-mach-number area) becomes large, a low-energy area, in which the speed is low, becomes large and a loss area becomes large. From this drawing, it is understood that the loss area becomes small in the right drawing.

5 [0035]

Figs. 7A to 7C show the third embodiment in which the blade row according to the invention is applied to a rotor blade row. In this drawing, Fig. 7A is a schematic side view showing a rotor blade row 20, Fig. 7B is a
10 sectional view taken along the line A-A, and Fig. 7C is a sectional view taken along the line B-B.

[0036]

In Fig. 7A, the rotor blade row 20 according to the invention is formed by plural main rotor blades 22 and
15 plural sub-rotor blades 24. In this drawing, each sub-rotor blade 24 is located on the rear side of each main rotor blade 22.

The plural main rotor blades 22 are located in a circumferential direction of the rotary axis Z-Z of the
20 rotor blade row so as to have an interval therebetween. Additionally, the plural sub-rotor blades 24 are located between the main rotor blades 22 so as to have an interval therebetween in a circumferential direction. Accordingly, the number of the main rotor blades 22 is the same as that
25 of the sub-rotor blades 24.

[0037]

The main rotor blade 22 is formed by a basic blade

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portion 22a which has the same shape as that of the sub-rotor blade 24 and a forward blade portion 22b which extends to the upstream side of the basic blade portion. Accordingly, the basic blade portion 22a of the main rotor blade has the same configuration as that of the sub rotor blade 24 except for the existence of the forward blade portion 22b.

[0038]

The basic blade portion 22a of the main rotor blade 22 and the sub-rotor blade 24 are located at the same position in an axial direction, and a basic rotor blade row is formed therebetween. In this basic rotor blade row, it is desirable to have a uniform circumferential interval between the basic blade portion 22a and the sub-rotor blade 24.

[0039]

The forward blade portion 22b of the main rotor blade 22 forms a forward rotor blade row which is formed in the vicinity of at least a radial inner end (on a hub side) so as to have a circumferential interval larger than that of the basic rotor blade row 22a. The circumferential interval of the forward rotor blade row is approximately two times that of the basic rotor blade row.

[0040]

Figs. 8A to 8C are views showing the fourth embodiment in which the blade row according to the invention is applied to the rotor blade row. In this

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drawing, Fig. 8A is a schematic side view showing the rotor blade row 20, Fig. 8B is a sectional view taken along the line A-A, and Fig. 8C is a sectional view taken along the line B-B.

5 In this example, a front edge 22c of the main rotor blade 22 is located on the downstream side of a front edge 24c of the sub-rotor blade 24 from a radial middle portion to an outer end.

10 The other configurations are the same as those of the third embodiment.

[0041]

15 According to the above-described configuration, the rotor blade row 20 is formed by the basic rotor blade row which is formed by the basic blade portion 22a of the main rotor blade 22 and the sub-rotor blade 24 and the forward rotor blade row which is formed by only the forward blade portion 22b of the main rotor blade 22. The number of blades of the forward rotor blade row is smaller than that of (is a half of) the basic rotor blade row. Accordingly, 20 it is possible to reduce the fluid friction loss of the blade portion and to efficiently increase the pressure.

[0042]

25 Since the circumferential interval of the forward rotor blade row in the vicinity of the radial inner end is larger than that of the basic rotor blade row (by approximately two times), it is possible to expect a wide dynamic range, high efficiency, and an expansion of a

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throat area on the hub side determined by the interval of the forward blade row.

[0043]

With the configuration in which the front edge 22c of the main rotor blade 22 is located on the downstream side of the front edge 24c of the sub-rotor blade 24 from the radial middle portion to the outer end (the fourth embodiment), the circumferential interval of the front edge of the sub-rotor blade 24 on the tip side is large (by approximately two times). Accordingly, it is possible to expand the throat area at the tip side and to expect the pressure loss reduction at a high-ratio flow rate.

In addition, it is possible to realize a decrease in weight as a whole as much as the short sub-rotor blade on the hub side.

[0044]

Therefore, according to the invention, in any case of the stator blade row 10 and the rotor blade row 20, it is possible to reduce pressure loss of the compressor, and to more increase an air flow rate while maintaining a compression characteristic than that of the conventional art.

[0045]

Furthermore, the invention is not limited to the above-described embodiments, but may be, of course, modified into various forms without departing from the spirit of the invention.

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CLAIMS

1. A blade row of an axial flow type compressor in which a rotor blade row and a stator blade row are
5 alternately arranged in an axial direction,

wherein the stator blade row is formed by plural main stator blades which are located in a circumferential direction of a rotary axis of the rotor blade row so as to have an interval therebetween and plural sub-stator blades
10 which are located between the main stator blades in a circumferential direction so as to have an interval therebetween,

wherein each main stator blade is formed by a basic blade portion which has the same shape as that of each sub-stator blade and a forward blade portion which extends to
15 the upstream side of the basic blade portion,

wherein the basic blade portion of the main stator blade and the sub-stator blade are located at the same position in an axial direction so as to form a basic stator
20 blade row therebetween, and

wherein the forward blade portion of the main stator blade forms a forward stator blade row which has a circumferential interval larger than that of the basic stator blade row in the vicinity of at least a radial inner
25 end.

2. A blade row of an axial flow type compressor in

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which a rotor blade row and a stator blade row are alternately arranged in an axial direction,

wherein the rotor blade row is formed by plural main rotor blades which are located in a circumferential direction of a rotary axis thereof so as to have an interval therebetween and plural sub-rotor blades which are located between the main rotor blades in a circumferential direction so as to have an interval therebetween,

wherein each main rotor blade is formed by a basic blade portion which has the same shape as that of each sub-rotor blade and a forward blade portion which extends to the upstream side of the basic blade portion,

wherein the basic blade portion of the main rotor blade and the sub-rotor blade are located at the same position in an axial direction so as to form a basic rotor blade row therebetween, and

wherein the forward blade portion of the main rotor blade forms a forward rotor blade row which has a circumferential interval larger than that of the basic rotor blade row in the vicinity of at least a radial inner end.

3. The blade row of the axial flow type compressor according to Claim 2, wherein a front edge of the main rotor blade is located on the downstream side of a front edge of the sub-rotor blade from a radial middle portion to an outer end.

Fig. 1
Prior Art

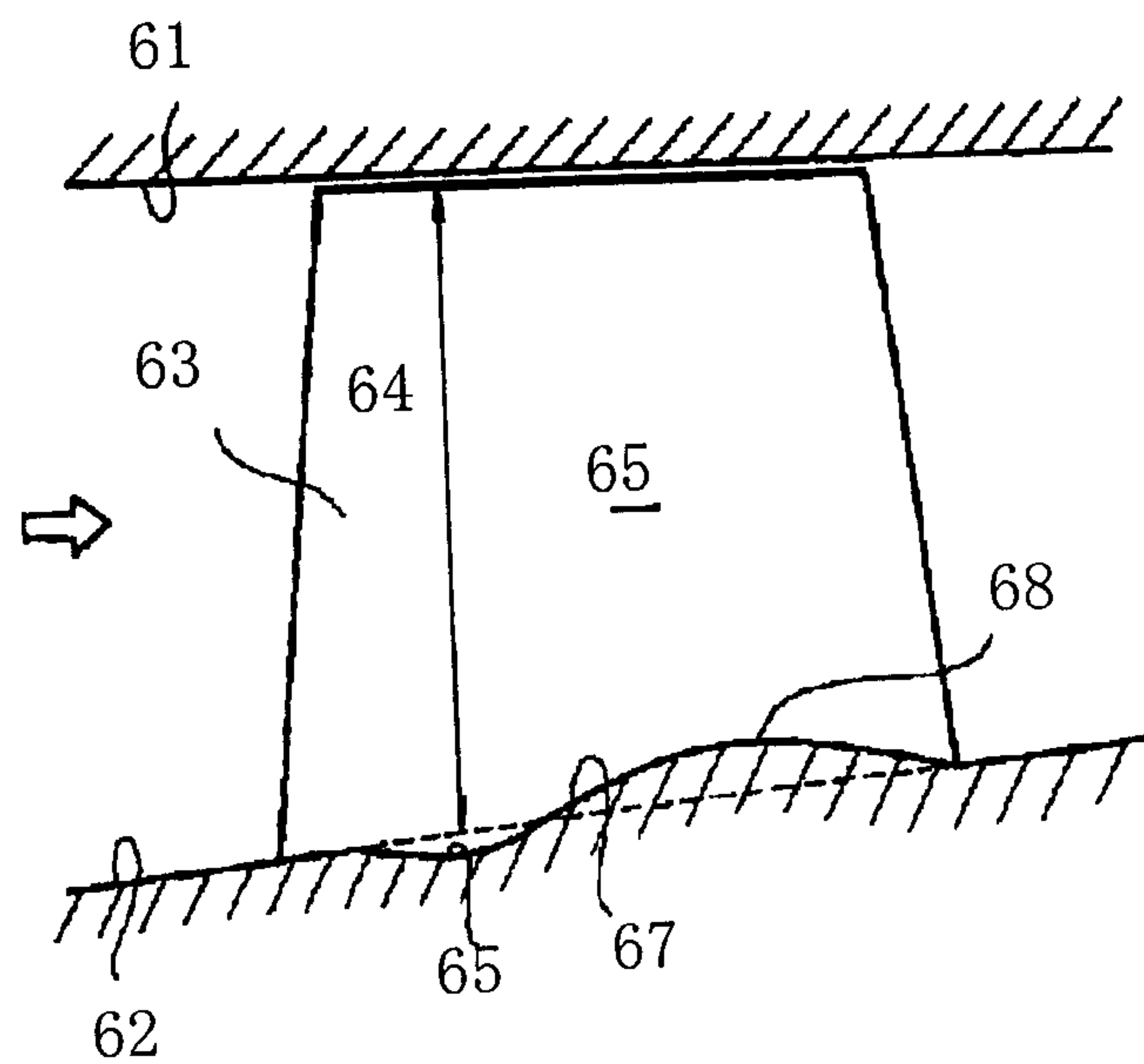


Fig. 2
Prior Art

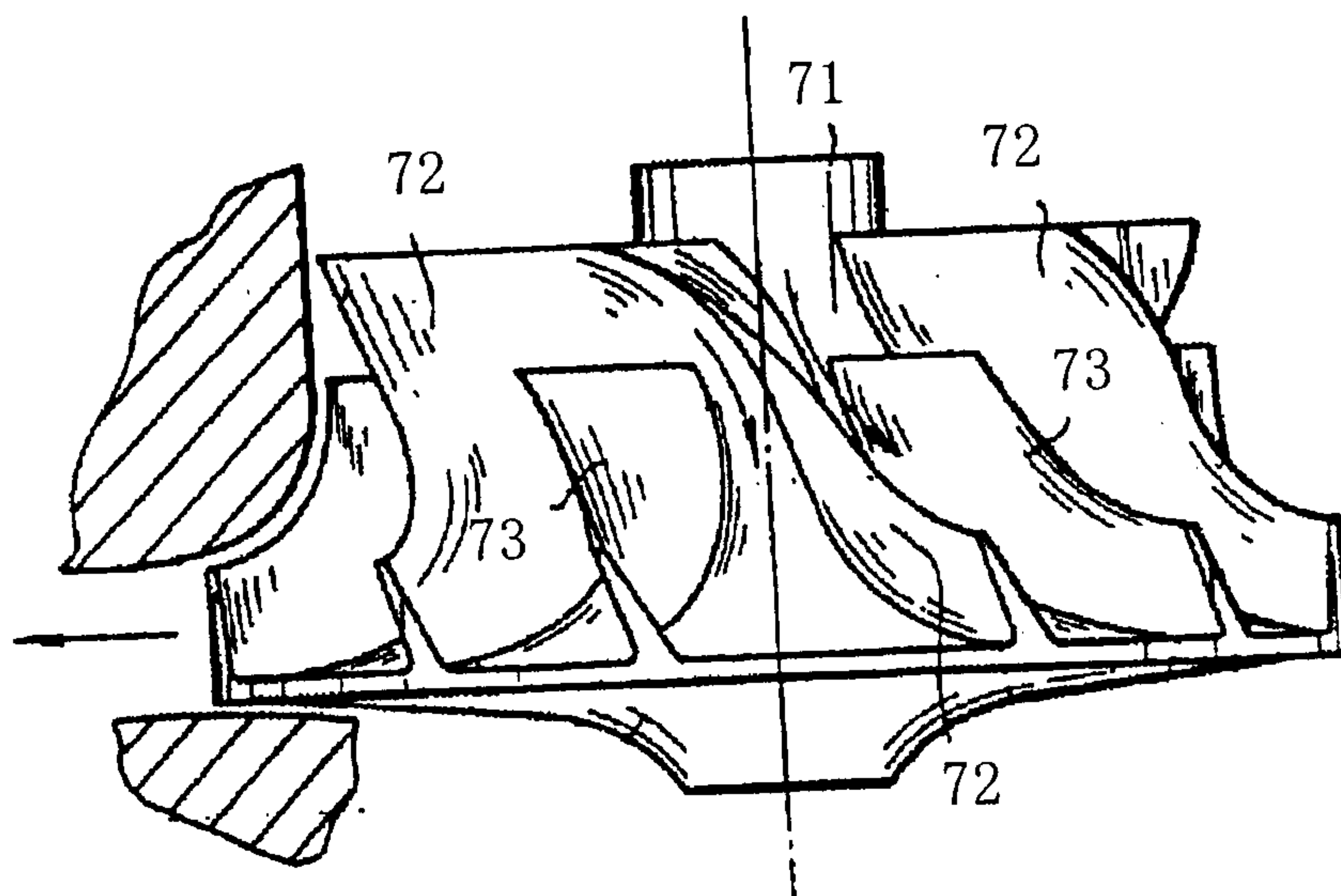


Fig. 3
Prior Art

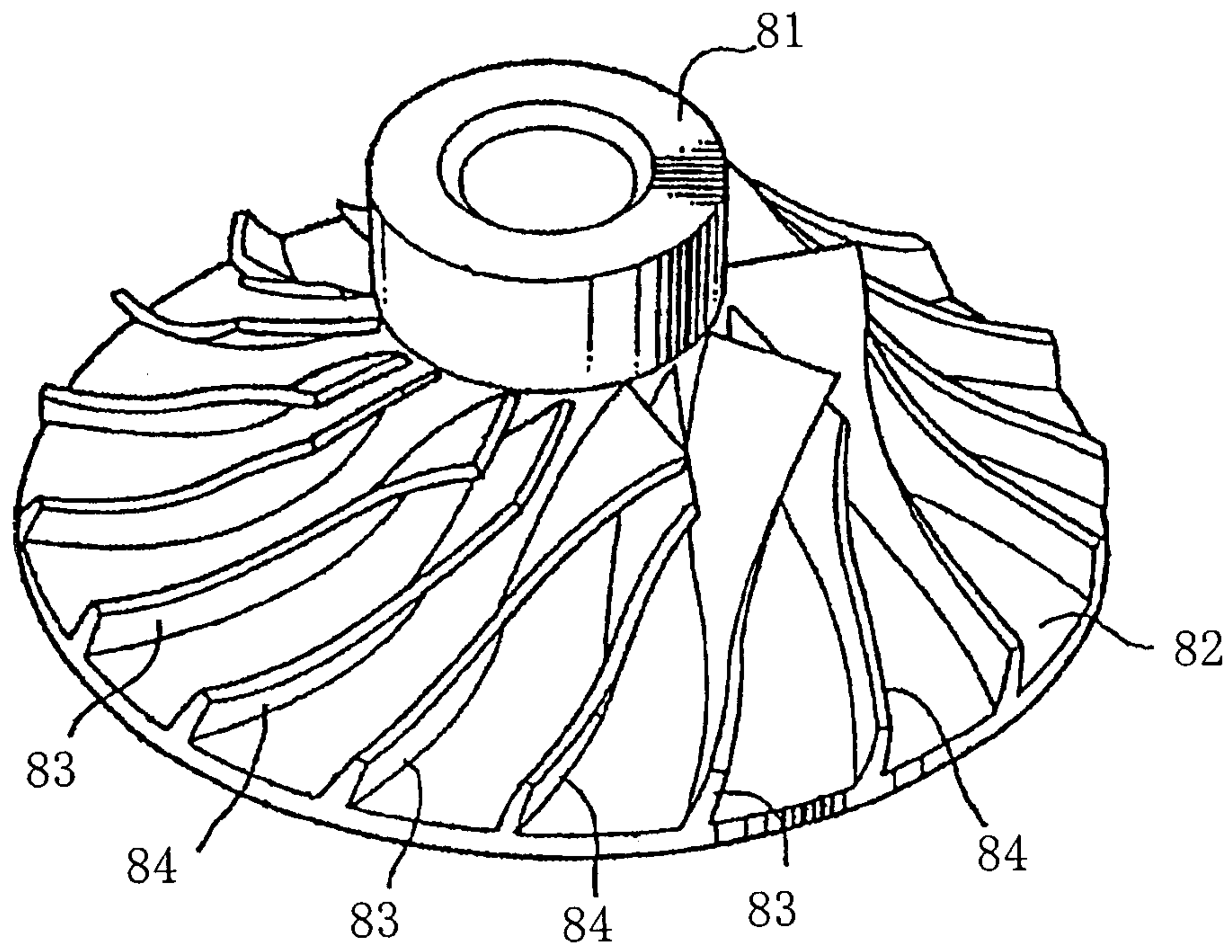


Fig. 4A

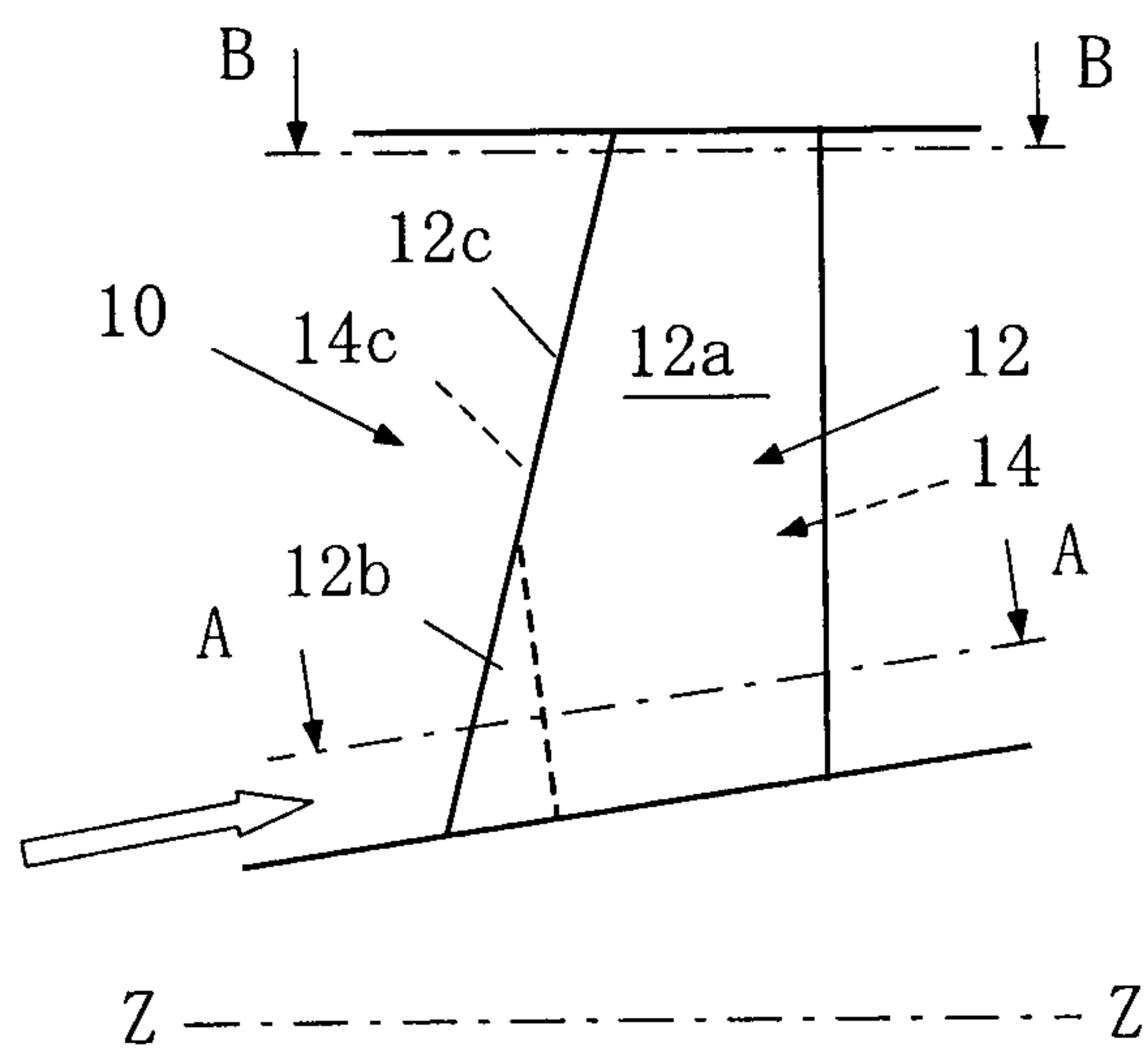


Fig. 4B

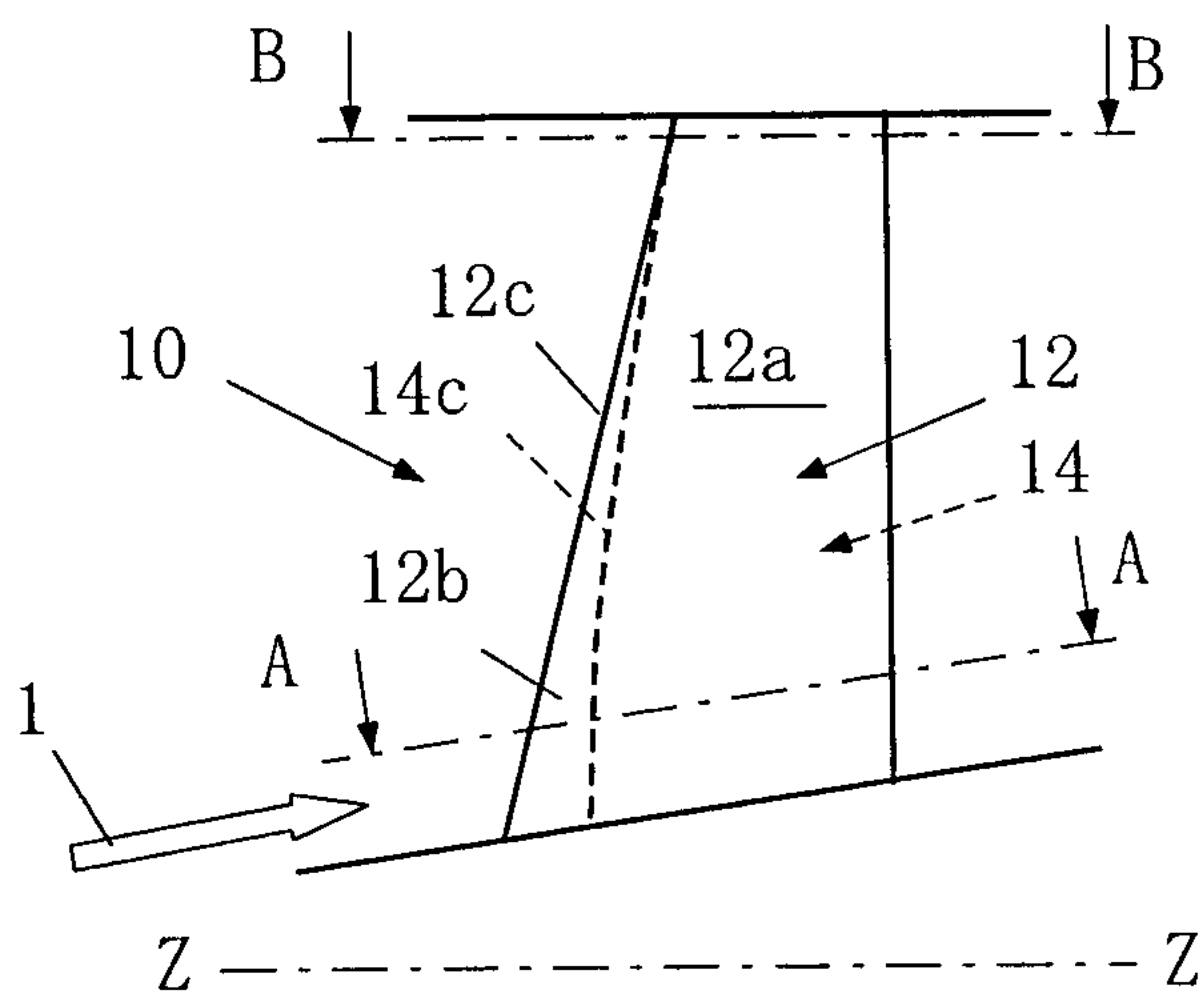


Fig. 4C

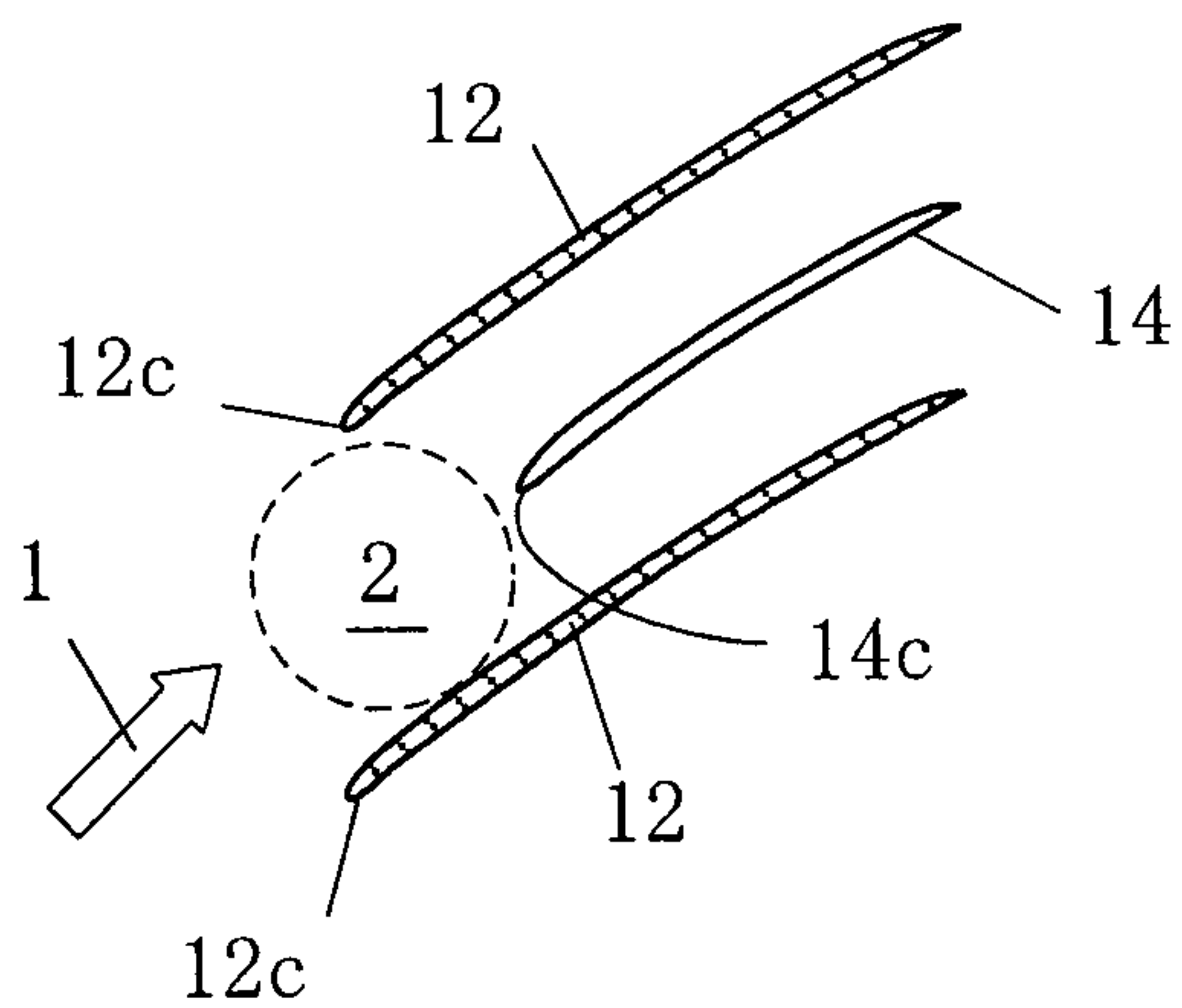
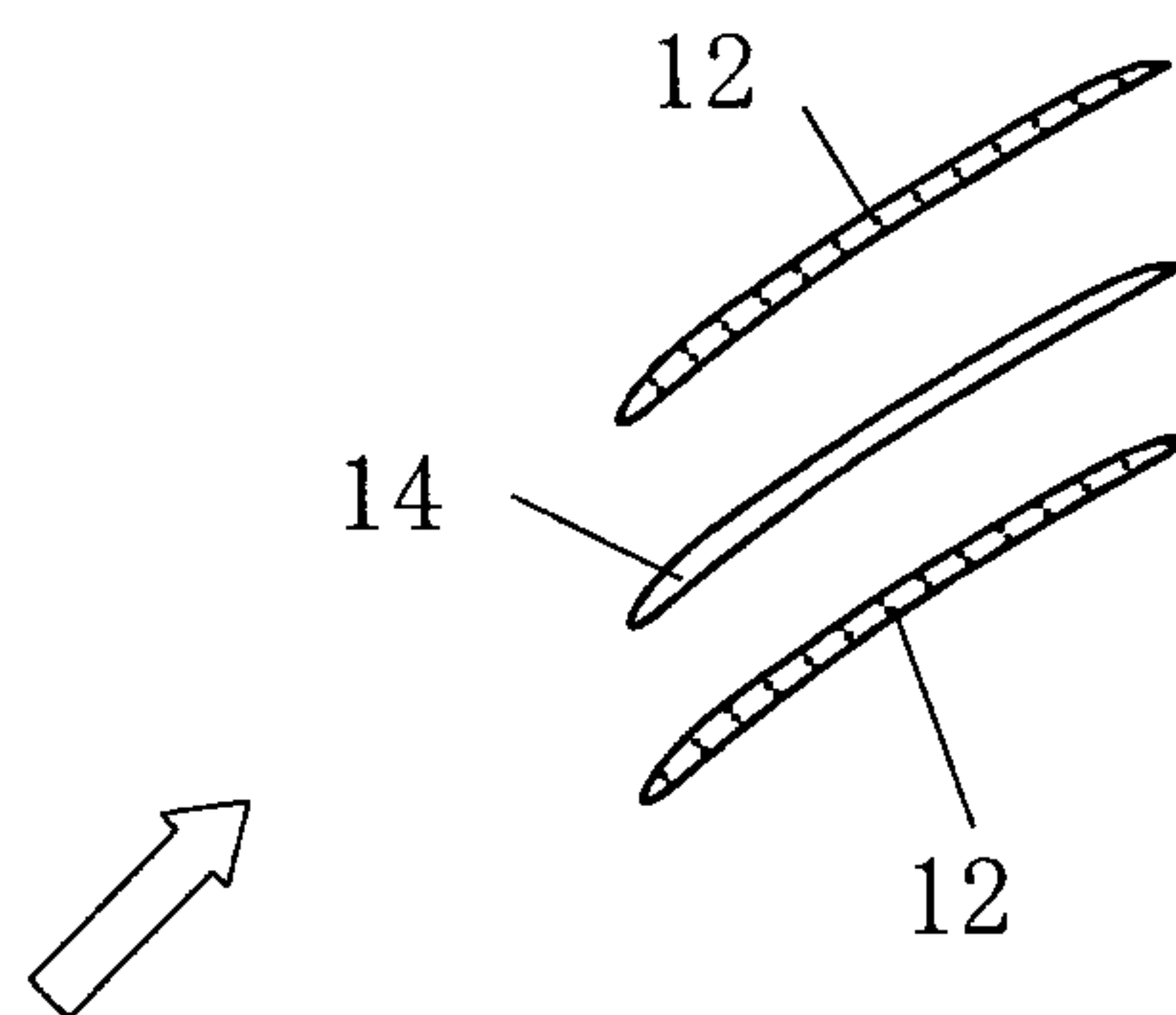


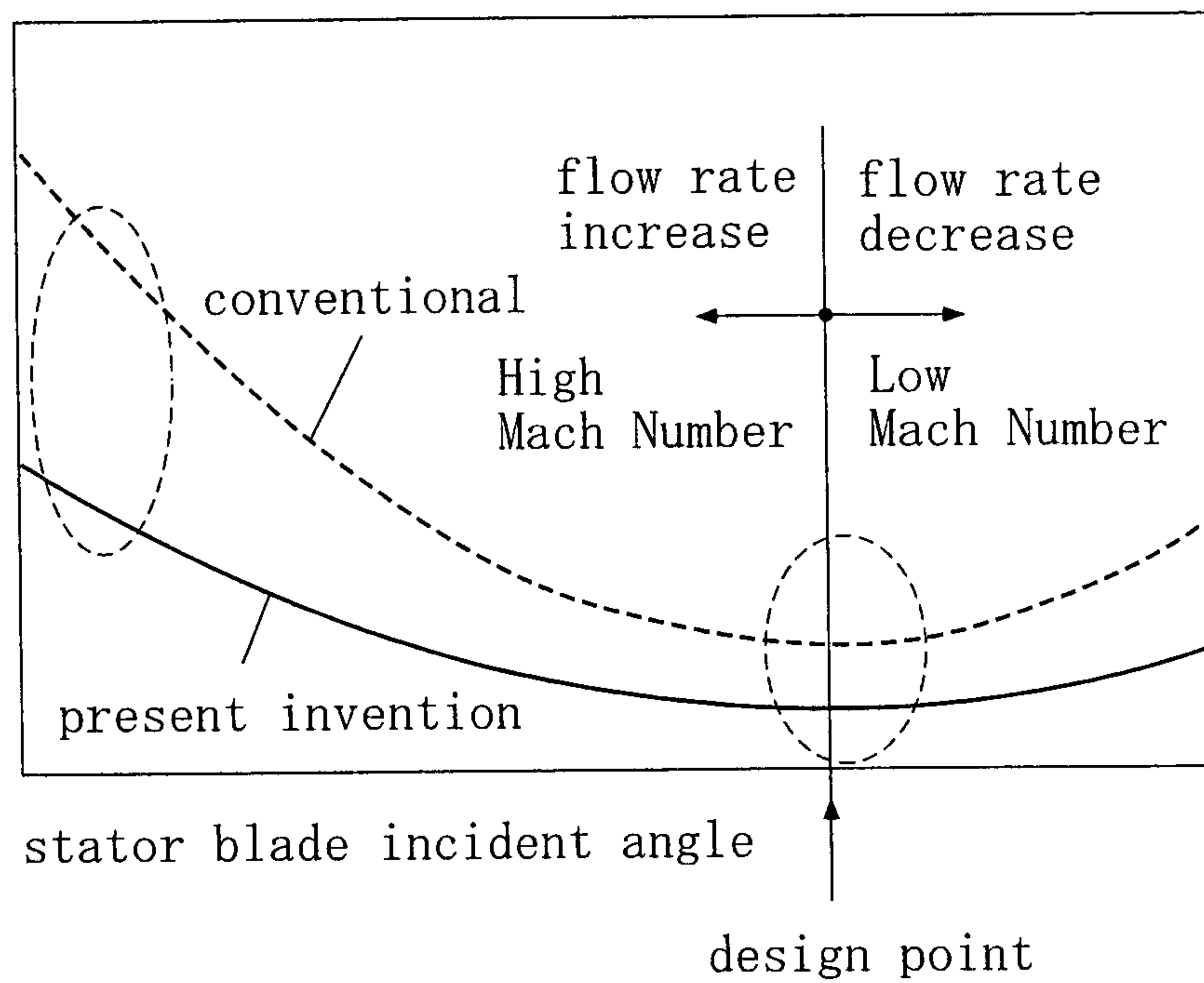
Fig. 4D



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Fig. 5

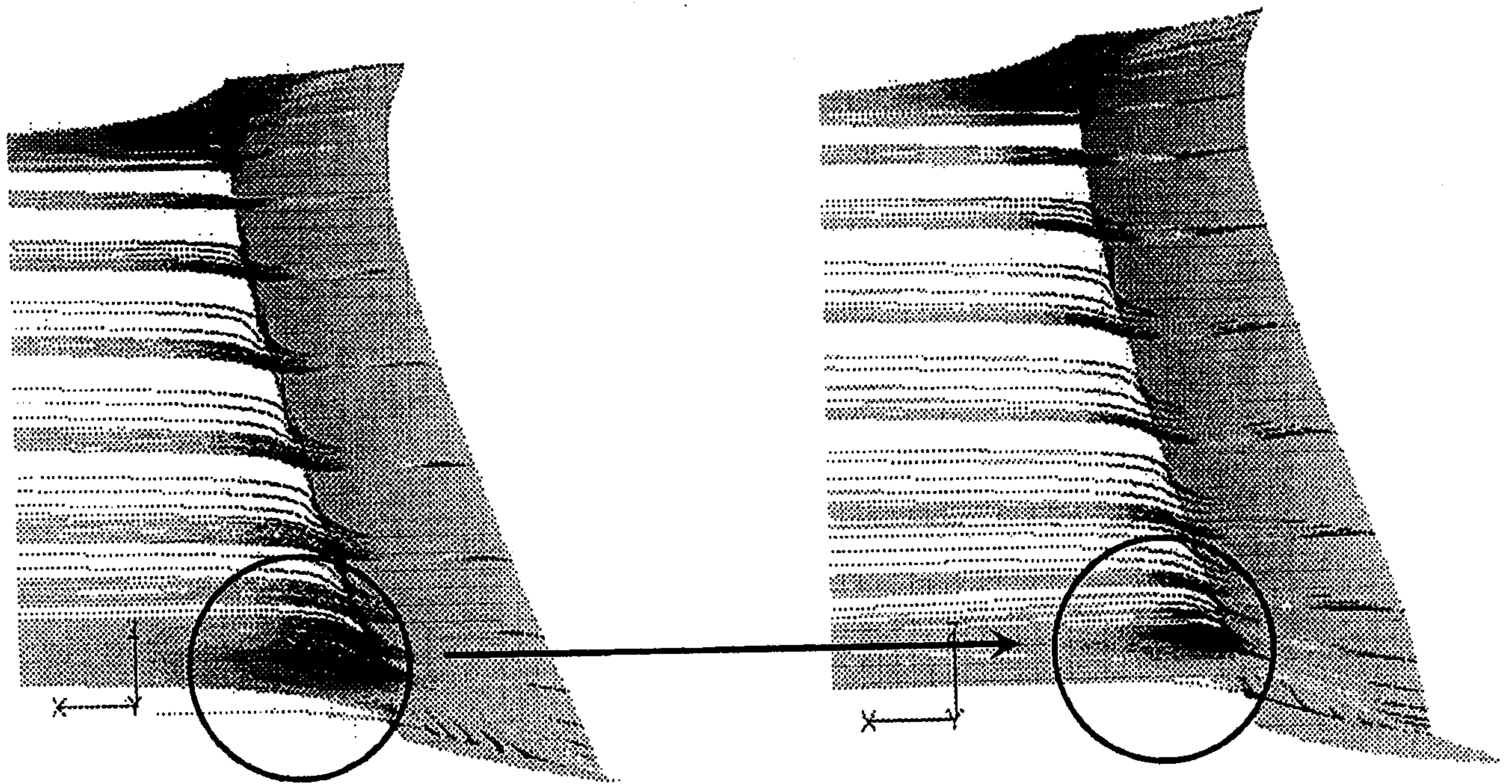
pressure loss
coefficient



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Fig. 6

streamlines of blade surfaces



Base Type

Invented Type

Fig. 7A

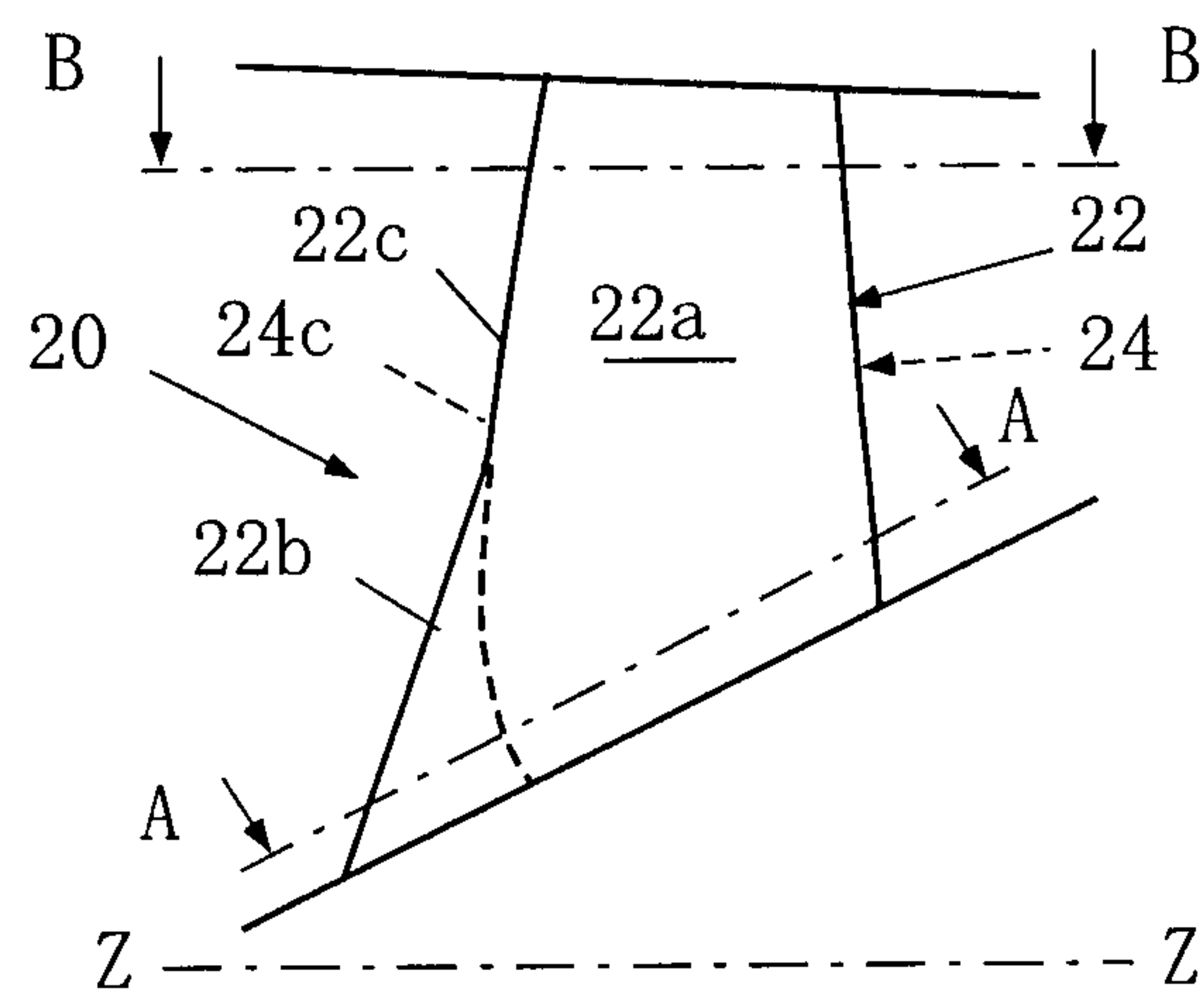


Fig. 7B

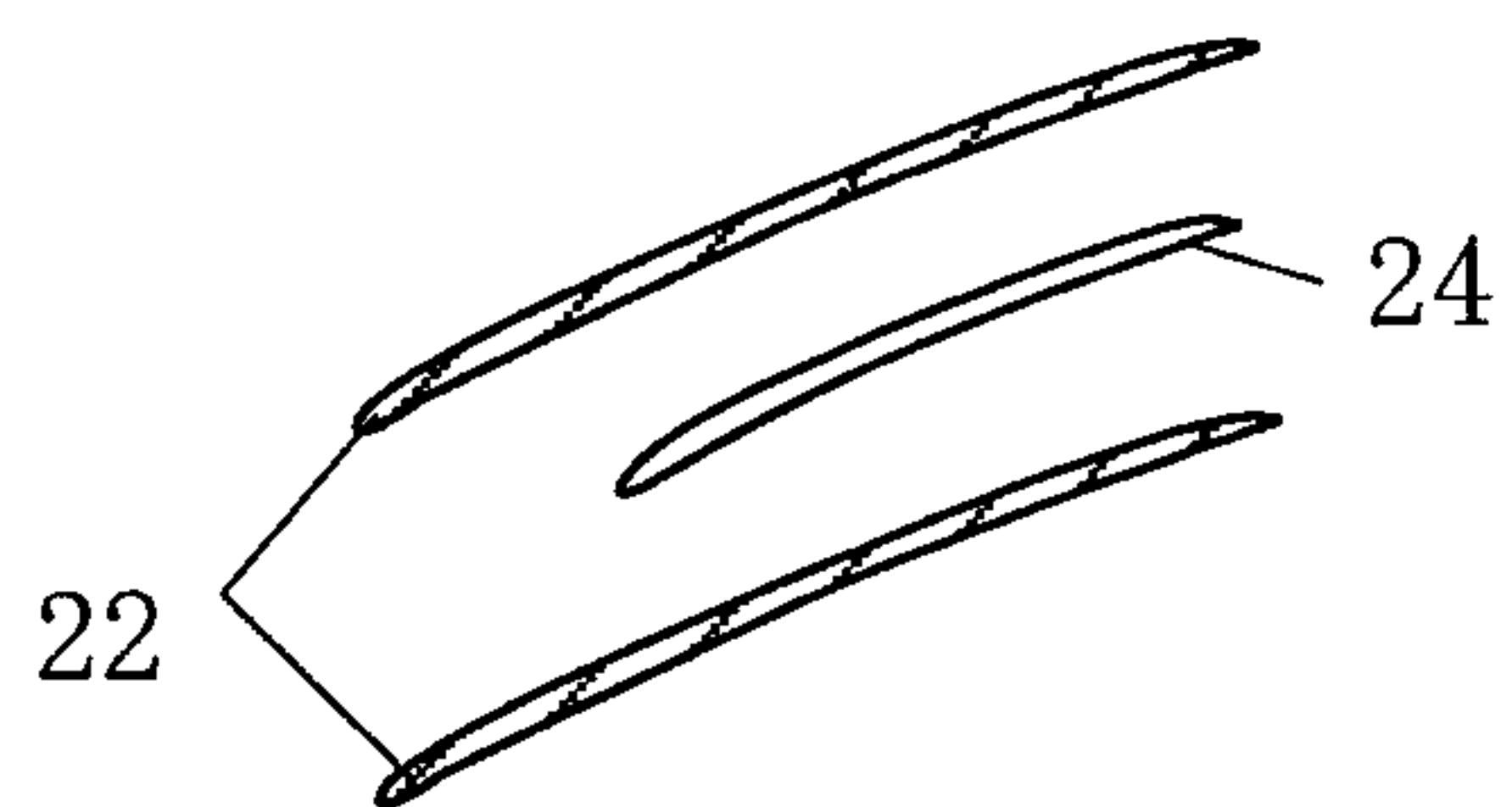


Fig. 7C

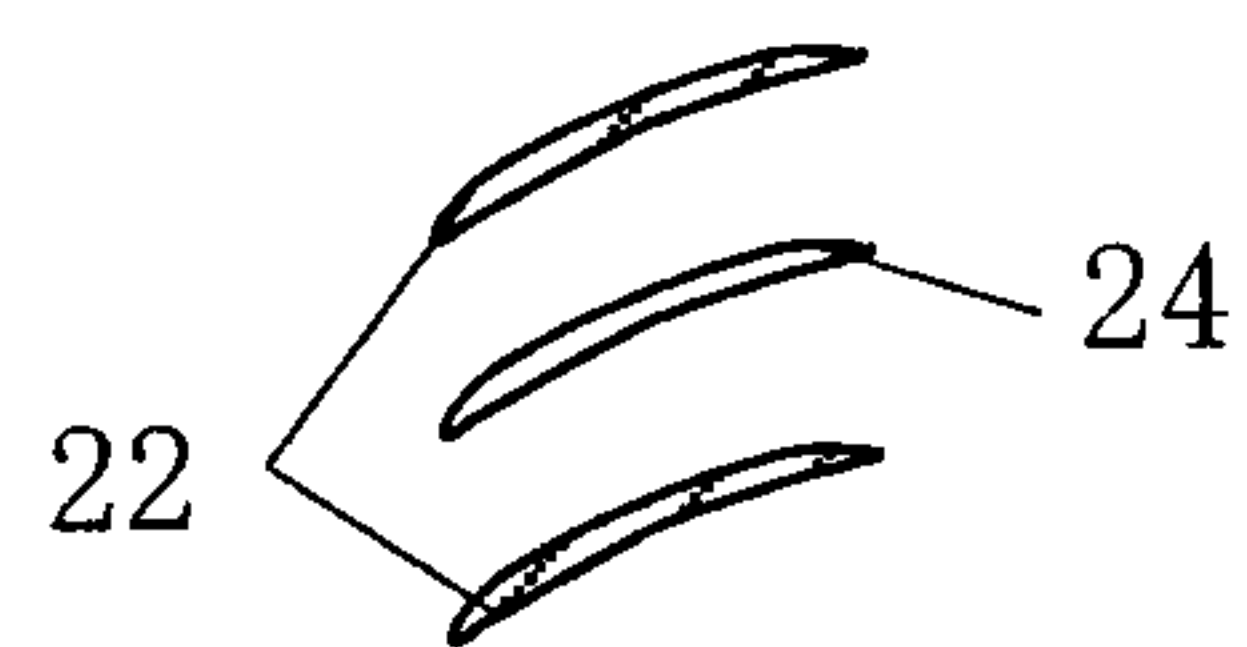


Fig. 8A

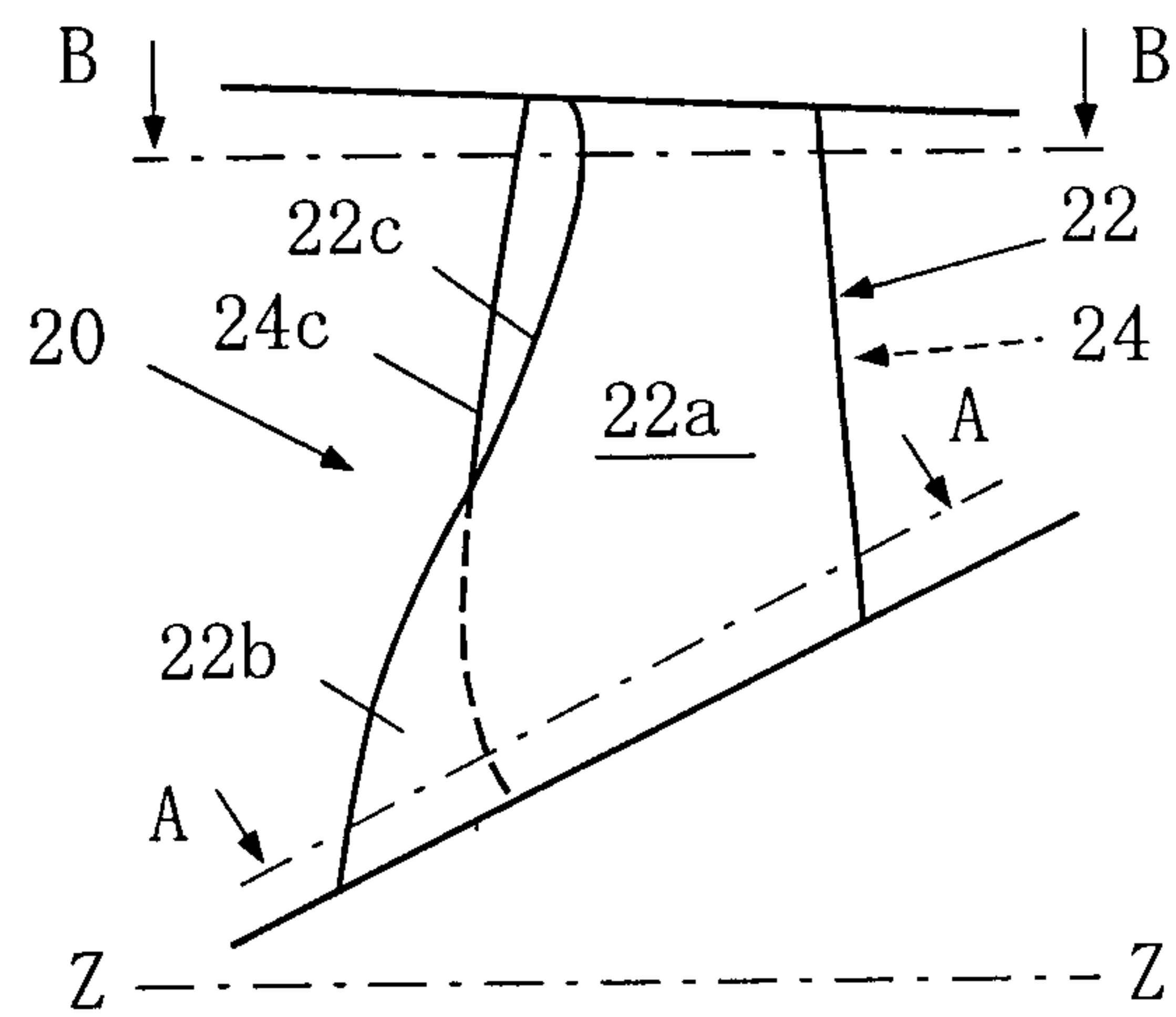


Fig. 8B

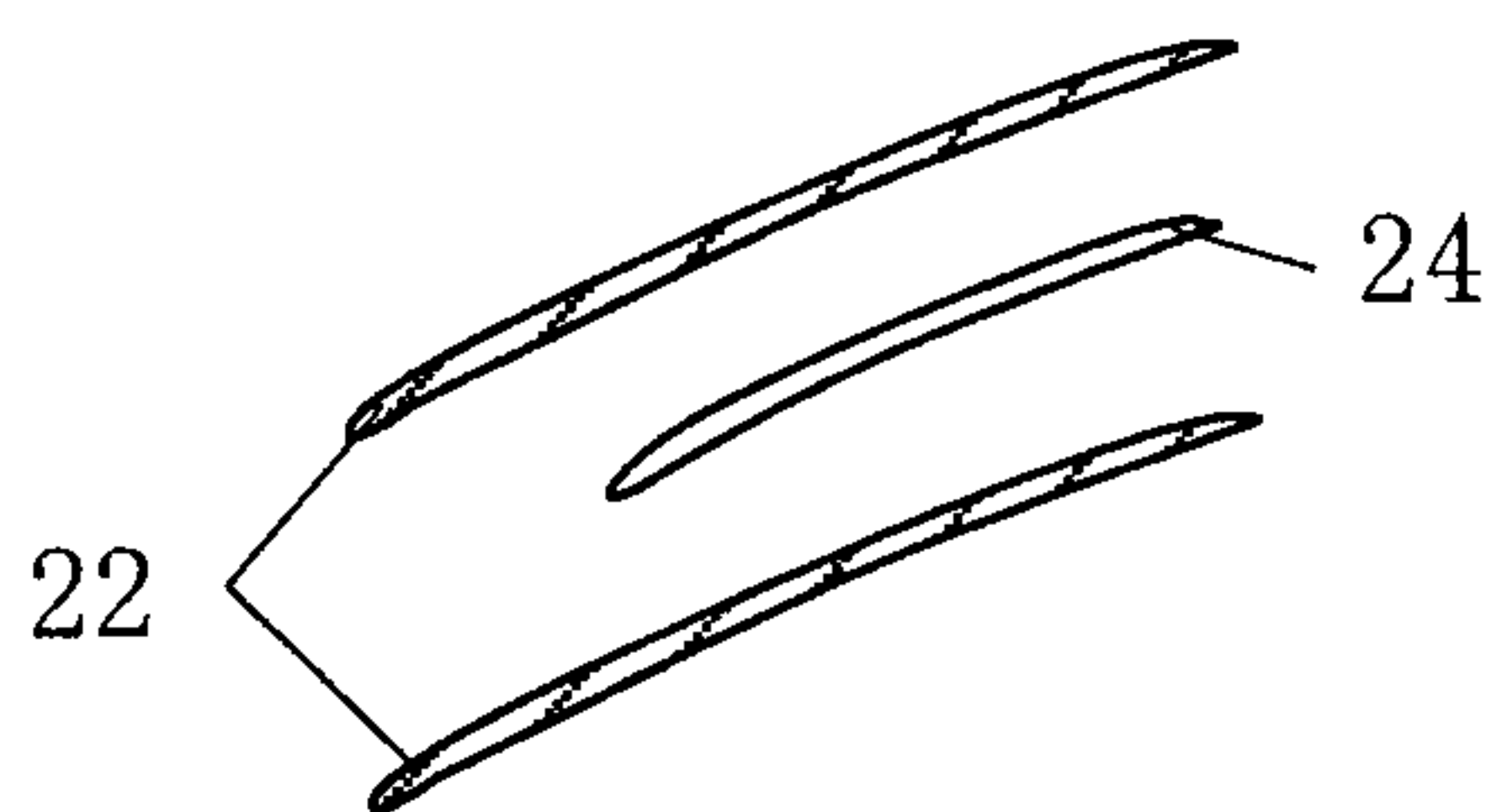


Fig. 8C

