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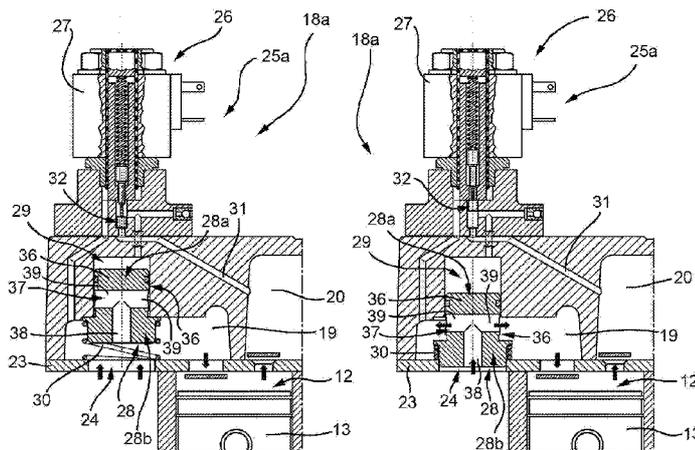


FIG. 4

FIG. 5

(57) **Abstract:** A compressor for a refrigerating plant comprising at least one cylinder (12); at least one piston (13), which slides alternately inside the cylinder (12); at least one head (18a) provided with a suction chamber (19), connected to a suction line (9) of the plant (1) and to the cylinder (12) to supply the cylinder (12) with a refrigerating fluid, and with a suction valve (25a), configured to regulate the flow rate of refrigerating fluid; the suction valve (25a) is movable between a first position, wherein is defined a first flow area which allows the suction of a first flow rate of refrigerating fluid, and a second position wherein is defined a second flow area smaller than the first flow area which allows the suction of a second flow rate of refrigerating fluid lower than the first flow rate.

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COMPRESSOR FOR A REFRIGERATING PLANT AND REFRIGERATING PLANT COMPRISING SAID COMPRESSOR

TECHNICAL FIELD

5 The present invention relates to a compressor for a refrigerating plant and a refrigerating plant comprising said compressor.

BACKGROUND ART

10 In the field of refrigerating plants there has always been a need for regulating the refrigerating capacity of the compressor based on the actual thermal load required. In fact, it often occurs that the refrigerating capacity of the compressor is excessive compared to the required thermal load. Such situation
15 occurs, for example, in supermarket refrigerating plants or refrigerating rooms during night hours. The thermal load required overnight is, in fact, definitely lower than the thermal load required during daily hours, wherein operations of massive removal of food from
20 refrigerating rooms, opening of freezer doors in a crowded place, etc. occur.

 If the refrigerating capacity of the compressor is constant, during night hours the compressor is capable of bringing the environment to the desired temperature
25 in a shorter time than the times required during daily

hours .

Such situation may determine the occurrence of various shortcomings, which contribute to compromising the efficiency of the entire refrigerating plant.

5 For example, the excessive refrigeration may lead to the formation of ice on the evaporator and determine a lowering of the dew point of the evaporator.

The lowering of the dew point may cause an excessive dehumidification of the processed air,
10 creating uncomfortable conditions if the compressor supplies an air-conditioning system, or may cause a significant weight drop of the product stored if the compressor supplies a refrigerating room.

Some known solutions provide for the switching off
15 of the compressor upon reaching the desired temperature and the restart of the compressor when the temperature exceeds a predetermined threshold value. Such solution, however, risks to dangerously increase the number of start-ups per hour of the compressor and significantly
20 reduce the operating life of the compressor.

Another known solution provides for a plurality of low-power compressors instead of a single high-power compressor. However, such solution, besides having a high starting cost and a more complicated control
25 system, does not overcome the disadvantage of having to

frequently restart the compressors to face the thermal load variation.

A further known solution provides for the complete closing of the suction channel of the refrigerating fluid of at least one of the pistons of the plurality of compressor pistons when a reduction of thermal load occurs. In particular, such solution provides the use of a solenoid valve, which closes the suction channel when the evaporation temperature and pressure decrease below a respective threshold value. In this way, the "pumping" effect of at least one of the plurality of compressor pistons is cancelled and the flow rate of the compressed refrigerating fluid is reduced, reducing the refrigerating capacity of the compressor. Such solution, however, allows to reduce the input flow rate by 50% at the utmost when the supply channel of one piston out of two is closed. This type of regulation is too rough and is not sufficient to optimize the performance and efficiency of the plant upon variation of the required thermal load.

In order to obviate this drawback, a control system of the solenoid valve is used which is configured to time the opening of the solenoid valve in order to obtain a desired reduction of the flow rate of refrigerating fluid input to the piston. A control

system of this type is described in the document US 2006/0218959. However, such solution proved to be not very efficient in that it determines a continuous fluctuation of pressures and temperatures inside the compressor, subjecting the inner parts of the compressor to dangerous stresses. Therefore, such solution is not reliable and requires constant monitoring of the operating conditions of the compressor.

DISCLOSURE OF INVENTION

10 It is therefore an object of the present invention to manufacture a compressor which is free from the prior art drawbacks mentioned above; in particular, it is an object of the present finding to manufacture a compressor reliable over time and the relative refrigerating capacity may be adjusted based on the thermal load in a simple and cheap way, but from the functional viewpoint and the construction viewpoint.

According to such objects, the present invention relates to a compressor for a refrigerating plant according to claim 1.

It is a further object of the finding to manufacture a refrigerating plant capable of adapting its own refrigerating capacity to the actual thermal load required.

25 According to such objects, the present invention

relates to a refrigerating plant according to claim 16.

BRIEF DESCRIPTION OF THE DRAWINGS

Further features and advantages of the present invention will appear clearly from the following description of a non-limiting embodiment example thereof, made with reference to the figures in the accompanying drawings, in which:

- figure 1 shows a schematic view of a refrigerating plant according to the present invention;
- 10 - figure 2 shows a side view, with sectional parts and parts removed for clarity, of a compressor for a refrigerating plant according to the present invention;
- figure 3 shows a front view, with sectional parts and parts removed for clarity, of the compressor of figure 15 2;
- figure 4 is a sectional view, with parts removed for clarity, of a first detail of the compressor of figure 2 in a first operating position;
- figure 5 is a sectional view, with parts removed for clarity, of a first detail of the compressor of figure 20 2 in a second operating position;
- figure 6 is a sectional view of a detail of a compressor for a refrigerating plant according to a second embodiment;
- 25 - figure 7 is a sectional view of a detail of a

compressor for a refrigerating plant according to a third embodiment;

- figure 8 is a sectional view of a detail of a compressor for a refrigerating plant according to a fourth embodiment ;

- figure 9 is a sectional view of a detail of a compressor for a refrigerating plant according to a fifth embodiment;

- figure 10 is a sectional view, with parts removed for clarity, of a second detail of the compressor for a refrigerating plant of figure 2 .

BEST MODE FOR CARRYING OUT THE INVENTION

In figure 1, reference numeral 1 indicates a refrigerating plant wherein a refrigerating fluid flows.

Refrigerating fluid means a refrigerating substance which may take the liquid or gaseous state inside the plant 1 depending on the pressure and temperature conditions to which it is subjected.

The refrigerating fluid is, for example, a fluid selected from the group consisting of HCFC, HFC, HFO, CO₂, HC.

The plant 1 comprises a compressor 2, a condenser 3, an expansion valve 4 and an evaporator 5.

A high-pressure delivery line 6 supplies the condenser 3 with the refrigerating fluid compressed by

the compressor 2. In particular, the refrigerating fluid supplied to the condenser 3 is in the form of vapour.

In the condenser 3 the refrigerating fluid in the form of vapour is transformed to the liquid form.

5 A high-pressure line 7 supplies the refrigerating fluid output from the condenser 3 to the expansion valve 4, where the pressure of the refrigerating fluid is lowered in order to lower the evaporation temperature.

10 A low-pressure line 8 supplies the low-pressure refrigerating fluid output from the expansion valve 4 to the evaporator 5, where heat is removed so that the refrigerating fluid evaporates at a constant pressure.

15 A low-pressure suction line 9 supplies the refrigerating fluid in the form of vapour and at low pressure to the compressor 2.

With reference to figure 2, the compressor 2 is an alternative semi-hermetic refrigerating compressor.

20 In the non-limiting example described and illustrated herein, the compressor 2 comprises a main body 10 inside which there are obtained two cylinder banks 11 having a substantially V-shape arrangement with each another (better shown in figure 3).

Each cylinder bank 11 comprises one or more cylinders 12.

25 In the example of the compressor described and

illustrated herein each cylinder bank 11 comprises two cylinders 12.

Inside each cylinder 12, a respective piston 13 moves alternately thanks to an electric motor 14.

5 In particular, each piston 13 is connected to a crankshaft 15 of the electric motor 14 by means of a rod-crank mechanism 16 (partially visible in the appended figures) and moves alternately in a suction direction A (figure 3) and in a compression direction C
10 (figure 3). In the suction direction A, the piston 13 draws the refrigerating fluid, while in the compression direction c, the piston 13 compresses the refrigerating fluid.

Each cylinder bank 11 is coupled to a respective
15 head 18 which puts the cylinders 12 in communication with the suction line 9 and the delivery line 6 of the plant 1.

With particular reference to figure 3, a first cylinder bank 11a is coupled to a first head 18a, while
20 a second cylinder bank 11b is coupled to a second head 18b.

Each head 18a, 18b comprises a low-pressure suction chamber 19 and a high-pressure delivery chamber 20.

The suction chamber 19 is in communication with the
25 suction line 9 of the plant 1 and with the cylinders 12

and is passed through by the refrigerating fluid during the stroke of the pistons 13 in the suction direction A .

The delivery chamber 20 is in communication with the delivery line 6 of the plant 1 and with the
5 cylinders 12 and is passed through by the refrigerating fluid during the stroke of the pistons 13 in the compression direction c .

In particular, the suction chamber 19 is preferably divided into two portion by a partition 23 provided with
10 an opening 24 through which the refrigerating fluid flows.

With reference to figure 2 and figure 4, the head 18a further comprises a suction valve 25a, which is arranged so as to regulate the flowing of the
15 refrigerating fluid drawn through the opening 24.

With reference to figure 10, the head 18b further comprises a suction valve 25b, which is arranged so as to regulate the flowing of the refrigerating fluid drawn through the opening 24 .

20 Figure 4 shows the head 18a provided with the suction valve 25a.

The suction valve 25a is movable between a first position, wherein the opening 24 is free and the regular flowing of the refrigerating fluid between the suction
25 line 9 and the cylinders 12 is allowed, and a second

position wherein the flow of refrigerating fluid between the suction line 9 and the cylinders 12 is reduced.

Preferably, in the first position the suction valve defines a first flow area of the refrigerating fluid coinciding with the flow area of the opening 24. The first flow area allows the flowing of a first flow rate of refrigerating fluid.

In the second position, the suction valve 25a defines a second flow area of the refrigerating fluid, which allows the flowing of a second flow rate of refrigerating fluid lower than the first flow rate.

In particular, the first flow area and the second flow area are shaped so as to allow, in the second position, the flowing of a second flow rate comprised between 10% and 90% of the first flow rate.

Preferably, in the second position there is a reduction of the flow rate of refrigerating fluid greater than or equal to 50%, with respect to the first position. It is understood that in the second position the suction valve 25a, in any case, allows the flowing of a flow rate of refrigerating fluid, which is minimum and greater than zero.

The first position and the second position are end positions of the suction valve 25a.

In the non-limiting example described and

illustrated herein, the suction valve 25a comprises a solenoid valve 26 provided with a coil 27, a shutter 28 movable in a seat 29, and a return spring 30 secured to the partition 23 and arranged around the shutter 28.

5 The seat 29 is in communication with the high-pressure delivery chamber 20 by means of a communication channel 31. In particular, the seat 29 is obtained in a wall of the suction chamber 19 and is arranged so as to be coaxial to the opening 24.

10 The solenoid valve 26 is provided with a closing member 32, which is coupled to a iron core 33 arranged inside the inner channel 34 of the coil 27 and a spring 35, arranged in the inner channel 34 in abutment against the iron core 33. The closing member 32 is arranged so
15 as to selectively occlude the communication channel 31.

In use, when the coil 27 is not powered with current, the spring 35 keeps the closing member 32 in the closing position of the communication channel 31.

20 When the coil 23 is powered with current, the iron core 33 is attracted to the coil 27 up to overcoming the force of the spring 35 and determines a displacement of the closing member 32 sufficient to free the communication channel 31.

25 When the communication channel 31 is free, the seat 29 is filled with the high-pressure refrigerating fluid

of the delivery chamber 20. The pressure of the refrigerating fluid overcomes the combined action of the return spring 30 and the suction pressure and pushes the shutter 28 towards the opening 24.

5 The shutter 28 comprises a main body 36 having a flow area substantially coinciding with the flow area of the opening 24 so as to completely engage the opening 24 in the second position of the valve 25a.

In particular, the shutter 28 comprises a main
10 portion 28a slidable inside the seat 29 and a head portion 28b, which has a radial dimension greater than the radial dimension of the main portion 28a. The head portion 28b is arranged in abutment against the wall of the suction chamber 19 wherein is obtained the seat 29
15 in the first position and it engages the opening 24 in the second position.

The main body 36 is provided with a channel 37 suitably shaped so as to allow the flowing of a determined flow rate of refrigerating fluid even when
20 the shutter 28 engages the opening 24.

Preferably, the channel 37 is obtained inside the shutter and is substantially T-shaped. In particular, the channel 37 comprises an inlet 38 facing the portion of the suction chamber 19 communicating with the low-
25 pressure suction line 9 and two outlets 39, each facing

a respective portion of the suction chamber 19 in the proximity of the respective cylinders 12 to be supplied.

Preferably, the main portion 28a comprises the outlets 39 while the head portion 28b comprises the
5 inlet 38.

In the non-limiting example described and illustrated herein, when the suction valve 25a is in the second position (configuration of figure 5) and the shutter 28 engages the opening 24, the channel 37
10 defines such a flow area that it allows the flowing of a flow rate equal to 50% of the first flow rate which normally flows through the opening 24 (configuration of figure 4 with the valve in the first position). In the practice, when the suction valve 25a is in the second
15 position, the flow rate of refrigerating fluid is reduced by 50%.

The reduction of the flow rate of refrigerating fluid supplied to the cylinders 12 determines a reduction of the refrigerating capacity of the
20 compressor 2.

The power supply to the coil 27 is preferably regulated by a control system (not illustrated in the appended figures), suitably configured to power the coil 27 when a reduction of the refrigerating capacity of the
25 compressor 2 is required.

As will be seen in detail hereinafter, the reduction of the refrigerating capacity of the compressor 2 is necessary when there is a reduction of the required thermal load.

5 Required thermal load herein and hereinafter means the refrigerating capacity to be supplied, or the heat to dissipate (expressed in kW), needed to keep a determined temperature in the refrigerating room or in the environment to air-condition.

10 Figure 6 shows a first variant of the present invention wherein the shutter 28 is provided with a plurality of channels 137 suitably shaped so as to define a flow area that allows the flowing of a determined second flow rate of refrigerating fluid.

15 Figure 7 shows a second variant, which envisages that the shutter 28 is shaped so that it does not entirely engage the opening 24 in the second position so as to define a flow area through which the flowing of a determined flow rate of refrigerating fluid is allowed.

20 Preferably, the flow rate of refrigerating fluid which flows through the flow area when the shutter 28 partially engages the opening 24 will be comprised between 10% and 50% of the flow rate which regularly flows through the free opening 24.

25 In the example of figure 7, the shutter 28 has a

flow area smaller than the flow area of the opening 24 to define such a flow area that it allows the passage of a determined flow rate of refrigerating fluid lower than the flow rate which regularly flows through the free opening 24 .

Figure 8 shows a third variant wherein the shutter 28 has a substantially truncated-cone shape. In this case, the controlled movement of the shutter 28 along a direction D allows to regulate the flow area defined by the shutter 28 and the opening 24. This embodiment of the shutter 28 allows a variable regulation of the flow rate of refrigerating fluid drawn. The closer the shutter 28 to the opening 24, the lower the amount of refrigerating fluid which passes through the opening 24.

A variant not shown envisages that the shutter has a pyramid or tapered shape .

A further embodiment illustrated in figure 9 envisages that the opening 24 is shaped so as to define a flow area for the refrigerating fluid even when the shutter 28 engages the opening 24. For example, the opening 24 is defined by two lobes and the shutter 28 is shaped so as to engage only one of the two lobes of the opening 24 .

With reference to figure 10, the suction valve 25b is arranged in the head 18b and is configured so as to

regulate the flowing of the refrigerating fluid drawn through the opening 24.

The suction valve 25b is movable between a third position, wherein the opening 24 is free and the regular
5 flowing of the refrigerating fluid between the suction line 9 and the cylinders 12 is allowed, and a fourth position wherein the flow of refrigerating fluid between the suction line 9 and the cylinders 12 is stopped. In the third position, the suction valve 25b defines a
10 third flow area of the refrigerating fluid, coinciding with the opening 24.

When the suction valve 25b is in the fourth position, there is a 100% reduction of the flow rate of refrigerating fluid.

15 In the non-limiting example described and illustrated herein, the suction valve 25b is substantially the same as the valve 25a with the exception that it comprises a shutter 128, which is configured so as to completely close the opening 24 when
20 the valve 25b is in the fourth position.

The shutter 128 comprises a main body 136 having a flow area substantially coinciding with the flow area of the opening 24 so as to completely engage the opening 24 in the fourth position of the valve 25a and to occlude
25 the flowing of the refrigerating fluid.

The main body 136 of the shutter 128 is full and is not provided with channels or openings capable of allowing the flowing of refrigerating fluid even when the shutter 128 engages the opening 24.

5 The power supply of the coil 27 of the suction valve 25b is preferably regulated by the control system (not shown in the appended figures) .

Similarly to what described above for the suction valve 25a, the control system is configured to power the
10 coil 27 when a reduction of the thermal load of the compressor 2 is needed.

In particular, the control system is configured to selectively activate the coil 27 of the suction valve 25a and the coil 27 of the suction valve 25b so as to
15 regulate the refrigerating capacity of the compressor based on the required thermal load.

In particular, the control system may selectively activate/deactivate the coils 27 of the suction valves 25a, 25b so as to substantially obtain four different
20 configurations:

- a nominal configuration wherein the suction valve 25a and the suction valve 25b are in the first and the third position, respectively, allowing the regular flowing of 100% of the refrigerating fluid drawn;
- 25 - a configuration at 75% wherein the suction valve

25b is in the third position and the coil 27 of the suction valve 25a is activated so that the suction valve 25a is in the second position and the shutter 28 engages the opening 24. In such configuration in the head 18a, there is a flow of refrigerating fluid of 25% of the total amount supplied by the compressor, while in the head 18b the suction valve 25b is in the third position and the flow of refrigerating fluid is regular and equal to 50% of the total amount supplied by the compressor. Accordingly, the flow of refrigerating fluid which reaches the cylinders 12 is equal to 75% of the flow of refrigerating fluid supplied in the nominal configuration;

- a configuration at 50% wherein the suction valve 27b is in the first position and the coil 27 of the suction valve 25b is activated so that the suction valve 25b is in the fourth position and the shutter 28 engages the opening 24. In such configuration, in the head 18a there is a regular flow of refrigerating fluid, while in the head 18b the opening 24 is closed and the flow of refrigerating fluid is null. Accordingly, the flow of refrigerating fluid which reaches the cylinders 12 is equal to 50% of the flow of refrigerating fluid supplied in the nominal configuration (only the refrigerating fluid drawn by

the head 18a is supplied) ;

- a configuration at 25% wherein the suction valve 25a and 25b are in the second and in the fourth position, respectively; in such configuration the shutter 28 in the head 18a allows the flowing of a flow of refrigerating fluid equal to about 25% of the regular flow, while the shutter 128 does not allow the flowing of refrigerating fluid in the head 18b.

It is understood that the configurations at 75% and 25% are exemplary and depend on the flow area defined by the suction valve 25a in the second position.

For example, the suction valve 25a is configured so as to define such a flow area that it allows a flow equal to 20% of the regular flow when it is in the second position, the configurations that may be obtained by means of the regulation carried out by the control system will be as follows: nominal configuration, configuration at 70%, configuration at 50% and configuration at 20%.

Advantageously, in the compressor according to the present invention it is possible to regulate the refrigerating capacity based on the required variations of thermal load in a simple and effective way.

In particular, the presence of at least one suction valve that allows the flowing of a minimum flow rate of

refrigerating fluid also in the second position allows to obtain greater flexibility in terms of regulation of the refrigerating capacity based on the thermal load.

Moreover, the construction of the suction valve 25b is
5 very simple and does not subject the compressor to pressure stresses or excessive vibrations which in the long term may jeopardize the reliability of the compressor itself.

Advantageously, the compressor according to the present
10 invention may be used in refrigerating plants that use any type of refrigerating fluid.

Finally, it is clear that changes and variations may be made to the compressor and plant described herein without departing from the scope of the appended claims.

CLAIMS

1. Compressor for a refrigerating plant comprising:
at least one cylinder (12) ; at least one piston (13) , which slides alternately inside the cylinder (12); at least one head (18a) provided with a suction chamber (19), connected to a suction line (9) of the plant (1) and to the cylinder (12) to supply the cylinder (12) with a refrigerating fluid, and with a suction valve (25a) , configured to regulate the refrigerating fluid flow rate;
the suction valve (25a) being movable between a first position, wherein is defined a first flow area, which allows the suction of a first flow rate of refrigerating fluid, and a second position wherein is defined a second flow area smaller than the first flow area which allows the suction of a second flow rate of refrigerating fluid lower than the first flow rate.

2. Compressor according to claim 1, wherein the first flow area and the second flow area are shaped so as to allow, in the second position, the flowing of a second flow rate comprised between the 10% and the 90% of the first flow rate.

3. Compressor according to claim 1 or 2, wherein the first flow area and the second flow area are shaped so as to allow, in the second position, the flow of a second flow rate equal to the 50% of the first flow rate.

4. Compressor according to any one of the foregoing claims, wherein the first position and the second position are end positions of the suction valve (25a).

5. Compressor according to any one of the foregoing claims, wherein the suction valve (25a) comprises a shutter (28) configured to engage an opening (24) of the suction chamber (19) in the second position; the shutter (28) and the opening (24) being shaped so as to define the first flow area in the first position and the second flow area in the second position.

6. Compressor according to claim 5, wherein the shutter (28) is shaped so as to define the second flow area to allow the flowing of the second flow rate of refrigerating fluid through the shutter (28).

7. Compressor according to claim 5 or 6, wherein the shutter (28) is provided with at least one channel (37) shaped so as to define the second flow area.

8. Compressor according to claim 7, wherein the channel (37) is in the shutter (28) and is preferably substantially T-shaped.

9. Compressor according to claim 5 or 6, wherein the shutter (28) is provided with a plurality of channels (137) shaped so as to totally define the second flow area.

10. Compressor according to claim 5, wherein the shutter (28) has a cross section smaller than the cross

section of the opening (24) so as to define the second flow area and allow the flowing of the second flow rate in the second position.

11. Compressor according to claim 5, wherein the shutter (28) is substantially truncated cone shaped and is configured so as to define the second flow area in the second position to allow the flowing of the second flow rate.

12. Compressor according to any one of the foregoing claims, comprising a control system configured to control the position of the suction valve (25a) on the basis of the required thermal load.

13. Compressor according to any one of the foregoing claims, comprising:
at least a further cylinder (12); at least a further piston (13), which slides alternately inside the further cylinder (12); a further head (18b) provided with a further suction chamber (19), connected to the suction line (9) of the plant (1) and to the further cylinder (12) to supply the further cylinder (12) with a refrigerating fluid, and with a further suction valve (25b), configured to regulate the refrigerating fluid flow rate;
the further suction valve (25b) being movable between a third position, wherein is defined a third flow area which allows the suction of a third flow rate of refrigerating

fluid, and a fourth position wherein the third flow area is occluded.

14. Compressor according to claim 13, wherein the further suction valve (25b) comprises a further shutter configured to engage a further opening (24) of the further suction chamber (19) in the fourth position; the further shutter (28) and the further opening (24) being shaped so as to occlude the third flow area in the fourth position.

15. Compressor according to claim 13 or 14, comprising a control system configured to control the position of the suction valve (25a) and of the further suction valve (25b) on the basis of the required thermal load.

16. Refrigerating plant comprising at least one suction line (9) and at least one compressor (2) as claimed in anyone of the foregoing Claims .

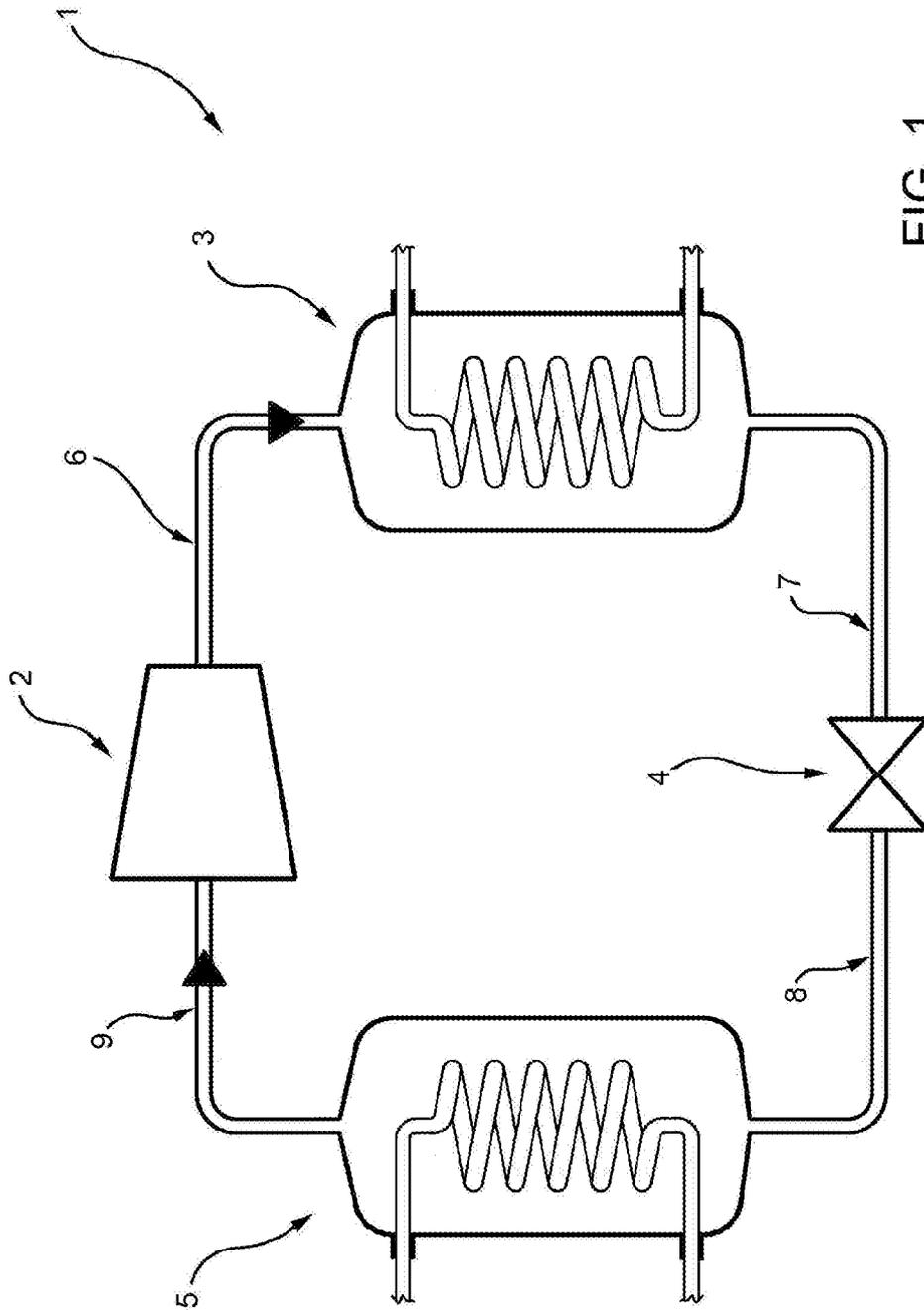
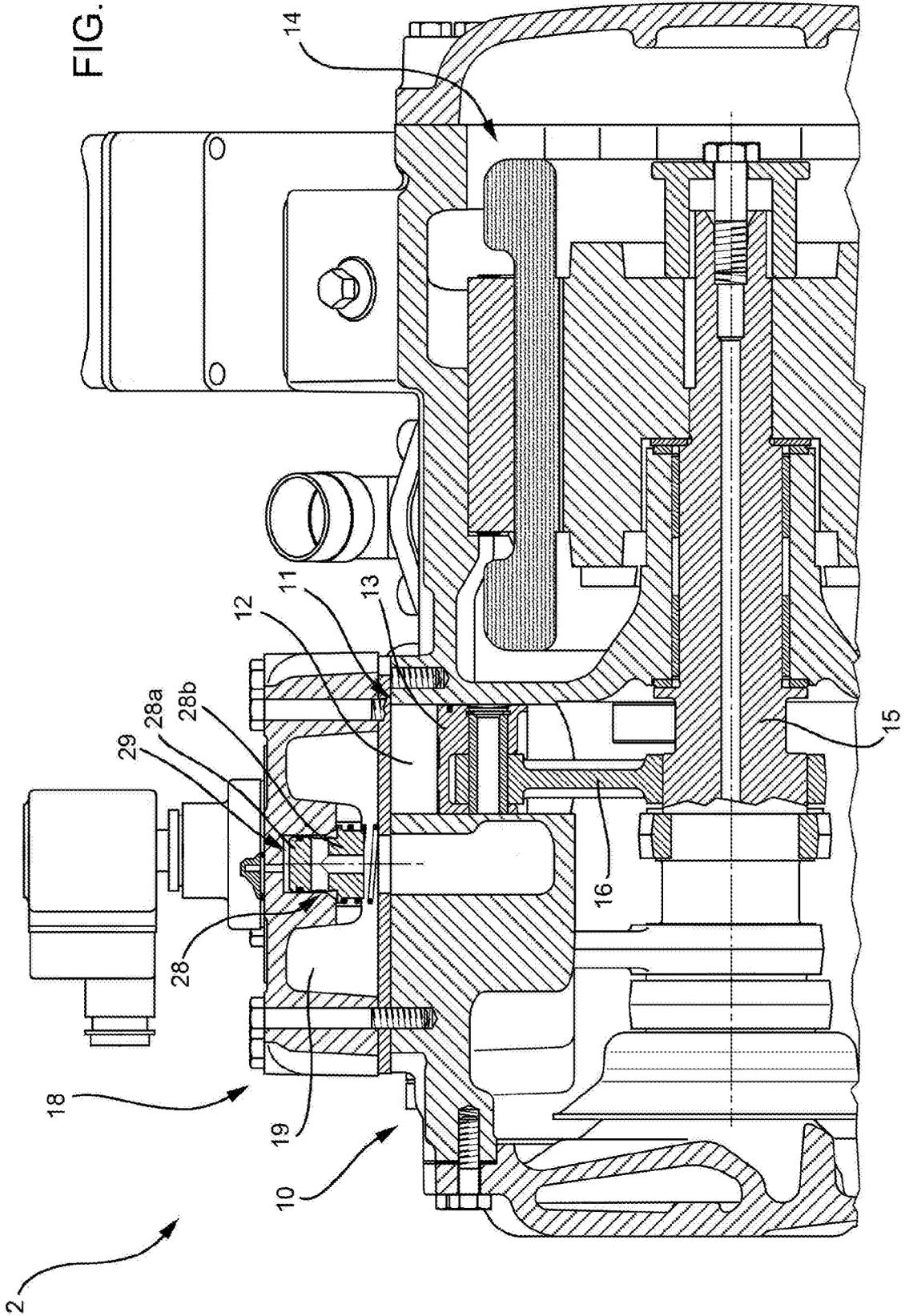


FIG. 1

FIG. 2



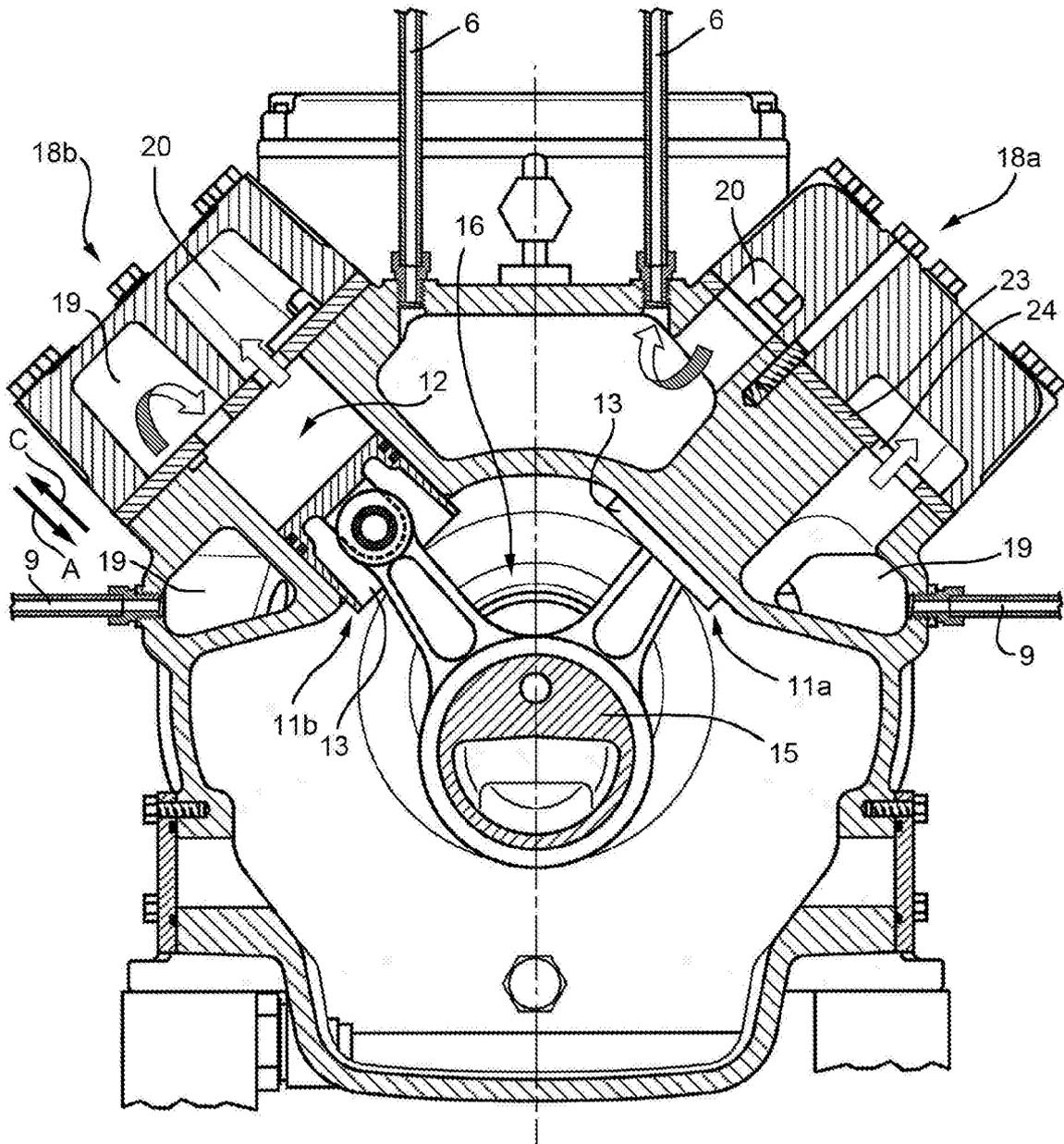


FIG. 3

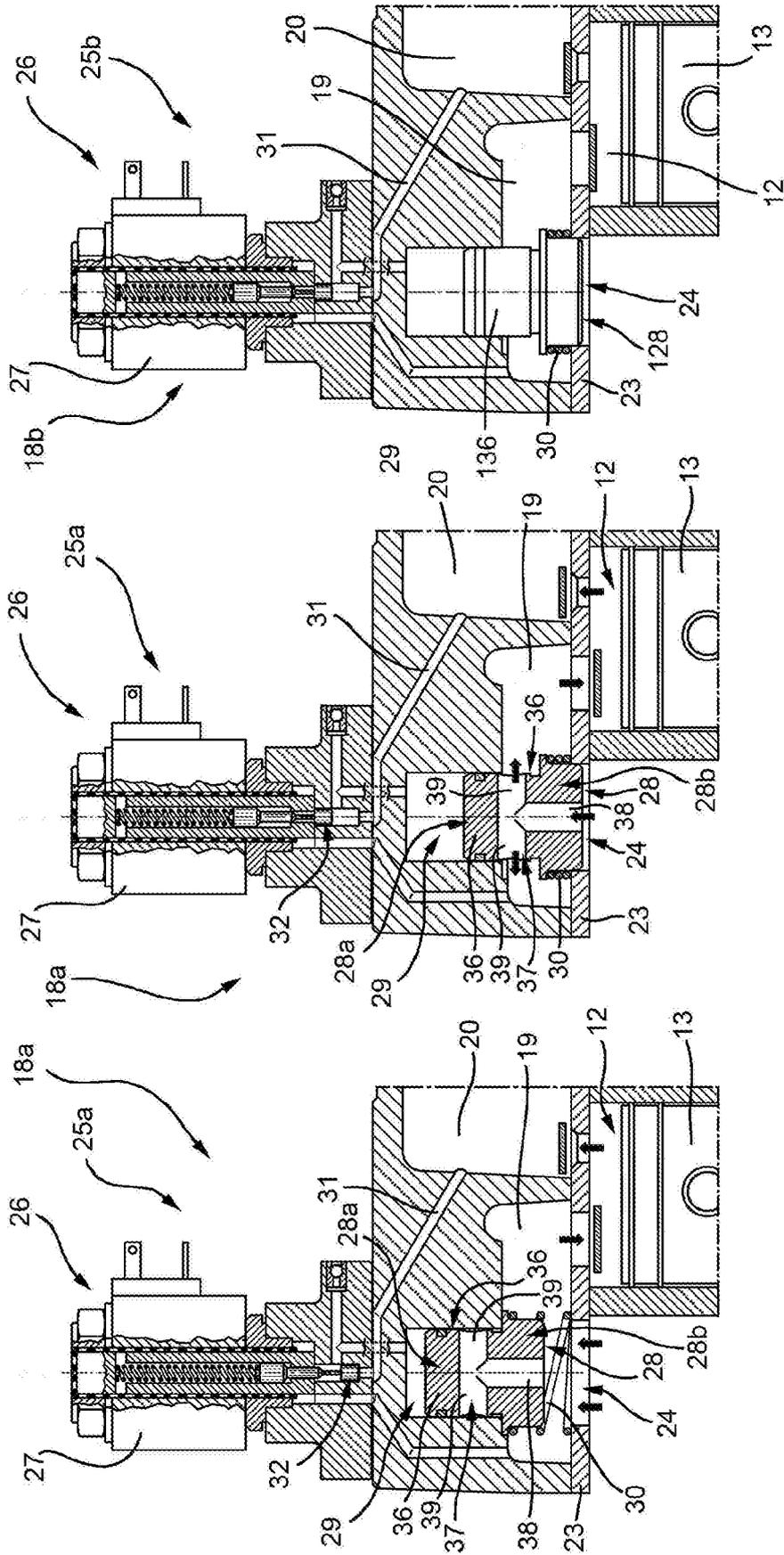


FIG. 5

FIG. 4

FIG. 10

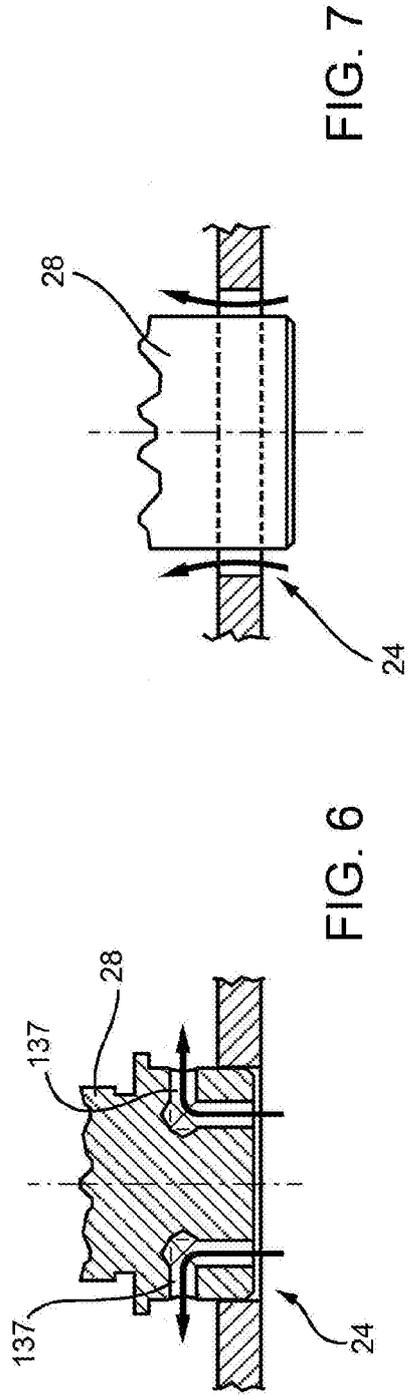


FIG. 7

FIG. 6

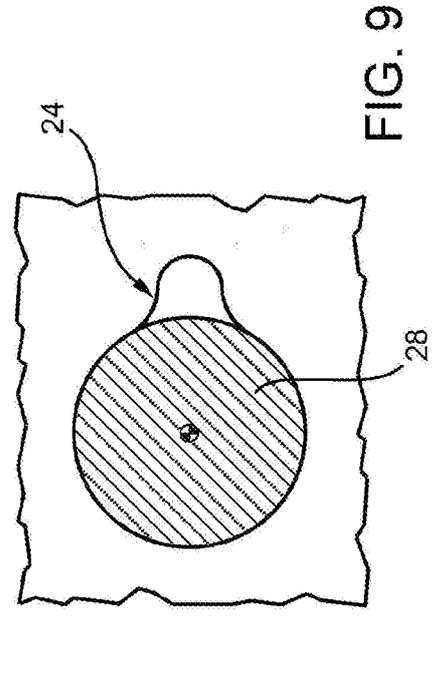


FIG. 9

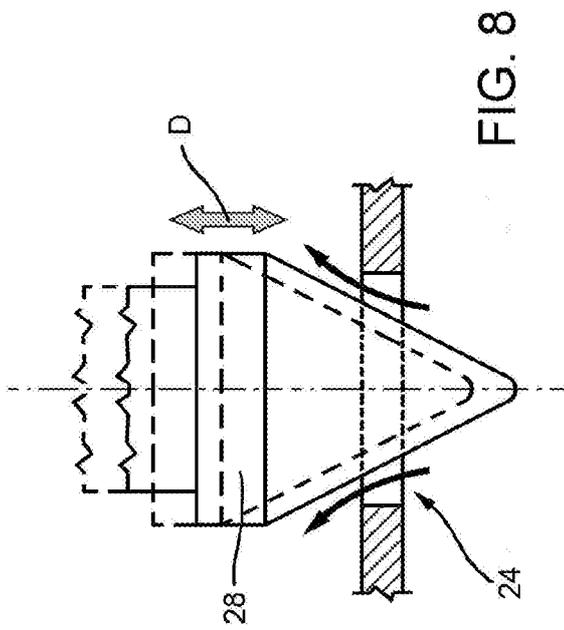


FIG. 8

INTERNATIONAL SEARCH REPORT

International application No PCT/IB2014/060655
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A. CLASSIFICATION OF SUBJECT MATTER
INV. F25B1/02 F25B41/04 F25B49/02
ADD.

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)
F25B F04B F16K

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)
EPO-Internal

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	EP 1 918 587 A2 (TOYOTA JIDOSHOKKI KK [JP]) 7 May 2008 (2008-05-07)	1-10, 12, 16
Y	paragraphs [0014] - [0039] ; figures 1-4 -----	11
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* Special categories of cited documents :

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