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Canadian Intellectual Property Office

CA 2972829 C 2022/03/08

(11)(21) **2 972 829**

(12) **BREVET CANADIEN**
CANADIAN PATENT

(13) **C**

(86) Date de dépôt PCT/PCT Filing Date: 2015/03/27
(87) Date publication PCT/PCT Publication Date: 2016/10/06
(45) Date de délivrance/Issue Date: 2022/03/08
(85) Entrée phase nationale/National Entry: 2017/06/30
(86) N° demande PCT/PCT Application No.: CA 2015/000187
(87) N° publication PCT/PCT Publication No.: 2016/154703

(51) Cl.Int./Int.Cl. *E21B 21/08* (2006.01), *E21B 1/28* (2006.01), *E21B 4/14* (2006.01)

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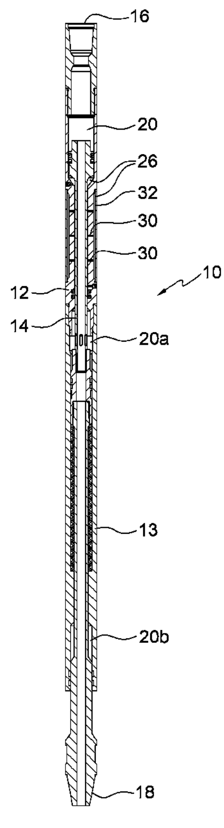
(54) Titre : APPAREIL ET PROCEDE DE MODIFICATION DE FORCE AXIALE
(54) Title: APPARATUS AND METHOD FOR MODIFYING AXIAL FORCE



(57) Abrégé/Abstract:
Embodiments disclosed herein relate to tools capable of amplifying or dampening axial forces produced by downhole equipment. More specifically, apparatus and methodologies provide a tool for imparting amplified axial loads (e.g. a hammer sub), or, in the alternative, for dampening/reducing downhole vibrations or "noise" (e.g. a suppressor sub).

(12) INTERNATIONAL APPLICATION PUBLISHED UNDER THE PATENT COOPERATION TREATY (PCT)

(19) World Intellectual Property Organization
International Bureau

(43) International Publication Date
6 October 2016 (06.10.2016)

WIPO | PCT

(10) International Publication Number
WO 2016/154703 A1

(51) International Patent Classification:
E21B 21/08 (2006.01) *E21B 4/14* (2006.01) *E21B 1/28* (2006.01)

(21) International Application Number: PCT/CA2015/000187

(22) International Filing Date: 27 March 2015 (27.03.2015)

(25) Filing Language: English

(26) Publication Language: English

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(81) Designated States *(unless otherwise indicated, for every kind of national protection available)*: AE, AG, AL, AM, AO, AT, AU, AZ, BA, BB, BG, BH, BN, BR, BW, BY, BZ, CA, CH, CL, CN, CO, CR, CU, CZ, DE, DK, DM, DO, DZ, EC, EE, EG, ES, FI, GB, GD, GE, GH, GM, GT, HN, HR, HU, ID, IL, IN, IR, IS, JP, KE, KG, KN, KP, KR, KZ, LA, LC, LK, LR, LS, LU, LY, MA, MD, ME, MG, MK, MN, MW, MX, MY, MZ, NA, NG, NI, NO, NZ, OM, PA, PE, PG, PH, PL, PT, QA, RO, RS, RU, RW, SA, SC, SD, SE, SG, SK, SL, SM, ST, SV, SY, TH, TJ, TM, TN, TR, TT, TZ, UA, UG, US, UZ, VC, VN, ZA, ZM, ZW.

(84) Designated States *(unless otherwise indicated, for every kind of regional protection available)*: ARIPO (BW, GH,

[Continued on next page]

(54) **Title**: APPARATUS AND METHOD FOR MODIFYING AXIAL FORCE

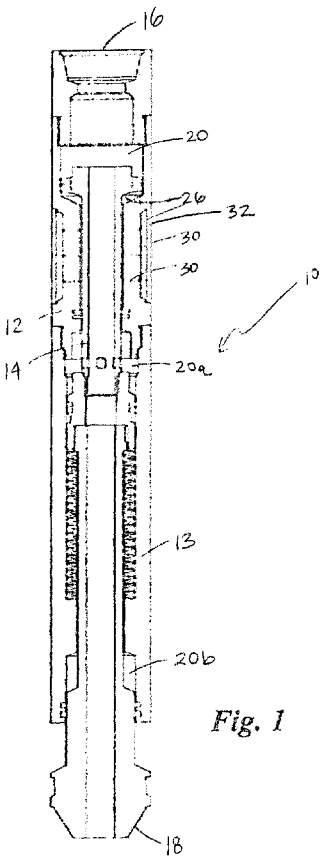

Fig. 1

(57) **Abstract**: Embodiments disclosed herein relate to tools capable of amplifying or dampening axial forces produced by downhole equipment. More specifically, apparatus and methodologies provide a tool for imparting amplified axial loads (e.g. a hammer sub), or, in the alternative, for dampening/reducing downhole vibrations or "noise" (e.g. a suppressor sub).

WO 2016/154703 A1

GM, KE, LR, LS, MW, MZ, NA, RW, SD, SL, ST, SZ, TZ, UG, ZM, ZW), Eurasian (AM, AZ, BY, KG, KZ, RU, TJ, TM), European (AL, AT, BE, BG, CH, CY, CZ, DE, DK, EE, ES, FI, FR, GB, GR, HR, HU, IE, IS, IT, LT, LU, LV, MC, MK, MT, NL, NO, PL, PT, RO, RS, SE, SI, SK, SM, TR), OAPI (BF, BJ, CF, CG, CI, CM, GA, GN, GQ, GW, KM, ML, MR, NE, SN, TD, TG).

Declarations under Rule 4.17:

— *as to applicant's entitlement to apply for and be granted a patent (Rule 4.17(ii))*

Published:

— *with international search report (Art. 21(3))*

APPARATUS AND METHOD FOR MODIFYING AXIAL FORCE

FIELD

[0001] Embodiments disclosed herein relate to tools capable of modifying (e.g., amplifying or suppressing) axial forces produced by downhole tools, and more specifically to tools for modifying the axial forces generated by downhole tools that impart movement of downhole equipment.

BACKGROUND

[0002] In the oil and gas industry, oil producers access sub-surface hydrocarbon-bearing formations by drilling long bore holes into the earth from the surface. Advances in drilling technologies have enabled the construction of deeper and longer wells. It is well known that downhole percussion tools can be used to enhance the rate of penetration in the drilling, to prevent buildup of friction due to pipe drag, or to increase the range in extended reach drilling operations.

[0003] Many downhole percussive tools, sometimes referred to as hammers or thrusters, are known to provide a pulsed fluid flow in order to increase the drilling rate. Pulsed fluid flow can be achieved by periodically restricting the drilling fluid flow through the tool, the restriction creating a pressure force which provides a percussive effect. In many tools, the percussive effect acts through a conventional shock sub, such that the cyclic fluid pressure causes the shock sub to extend or retract. However, such known percussive toolsare restricted in the size and frequency of axial force that they are capable of producing.

[0004] There is a need for a downhole percussion tool capable of amplifying axial force imparted onto downhole equipment with increased frequency (e.g., multiple “fires” per second continuously over extended periods of time). It is desirable that such an amplification tool may be used alone or in combination with known percussive tools, such as those tools described in United States Patent No. 8,167,051, United States Patent Application No. 13/381,297 or PCT/CA2014/000701, particularly in extended reach drilling operations (imparting loads on hundreds of meters of pipe), or difficult drilling operations (e.g., soft/hard formations). It is further desirable that, when implemented, such an amplification tool could reduce the need for downhole drilling jars.

SUMMARY

[0005] The present apparatus and methodologies combine mechanical and hydraulic processes to amplify axial forces generated by known percussion tools, as may be used in oil and gas drilling operations. It is an aspect of the present technology to achieve the amplified axial forces at a high rate of frequency over extended periods of time. According to embodiments herein, the present apparatus and methodologies may be utilized to amplify downhole percussion tools to increase the rate of penetration in drilling operations, to dislodge equipment that becomes stuck during drilling, as a hammer drill in extended reach operations, etc. It is appreciated that the present apparatus and methodologies may be used alone or in combination with other downhole equipment in stacked arrangement.

[0006] Broadly stated, the present apparatus for amplifying the transmission of fluid pressure into axial force may be adapted to permit the passage of fluid therethrough and may comprise: a first tubular housing having a sidewall forming a central housing bore capable of receiving the fluid, a second tubular piston, telescopically received within the housing bore, the piston having a sidewall forming a central piston bore, the piston bore being fluidically connected to the housing bore, and at least two first fluid chambers, each first fluid chamber formed between the housing and piston sidewalls and fluidically connected to the piston bore such that changes in fluid pressures within the at least two first fluid chambers are cumulative and induce axial movement of the tubular piston relative to the tubular housing.

[0007] In one embodiment, the present apparatus may further comprise at least one second fluid chamber disposed in between the at least two first fluid chambers operative to receive and resist opposed axial forces from the at least two first fluid chambers. Theat least one second chamber may comprise a fixed volume of fluid at a fixed pressure.

[0008] In other embodiments, the at least one second chamber may comprise pressure compensation means comprising:

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[0009] a plurality of radial fluid ports disposed through the housing sidewall for venting the fluid from the second fluid chamber through the housing sidewall, and

[00010] an annular membrane encircling the first tubular housing, for sealing the fluid ports and preventing fluid from exiting the tool.

[00011] Broadly stated, the present methodologies for amplifying the transmission of fluid pressure into axial forces may comprise: providing an amplification tool adapted to permit the passage of pressurized fluid therethrough, the tool having: a first tubular housing with a sidewall forming a central housing bore capable of receiving the fluid, a second tubular piston, telescopically received within the housing bore, the piston having a sidewall forming a central piston bore, the piston bore being fluidically connected to the housing bore, and at least two first fluid chambers, each first fluid chamber connected to the piston bore such that increases in fluid pressures within the at least two first fluid chambers accumulate to create sufficient axial forces to induce movement of the tubular piston relative to the tubular housing, providing a percussion tool, operatively connected to the amplification tool, and capable of generating axial force, utilizing the amplificationtool to amplify the axial load generated by the percussion tool. The method may further comprise that the second fluid chamber comprise pressure compensation means for receiving and resisting opposed axial forces from the at least two first fluid chambers.

[00012] Perhaps counterintuitively, in alternative and opposed operation, the present apparatus and methodologies may be used to magnify the dampening or suppression of “noise” vibrations produced by downhole equipment including, for example, pressure pulse frequencies impacting downhole Logging While Drilling (LWD) or Measurement While Drilling (MWD) tools. More specifically, without limitation, the present apparatusand methodologies may be configured to suppress or absorb larger axial loads or vibrations imparted on downhole equipment.

[00013] The above-mentioned and other features of the present apparatus and methodology will be best understood by reference to the following description of the embodiments.

BRIEF DESCRIPTION OF THE DRAWINGS

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[00014] Figure 1 is a cross sectional side view of a first embodiment of the present tool according to embodiments herein;

[00015] Figure 2 is a magnified cross sectional side view of the first embodiment of the present tool depicted in Figure 1;

[00016] Figure 3 is a perspective side view of the first embodiment of the tool depicted inFigure 1;

[00017] Figure 4 is a cross sectional side view of a second embodiment of the present tool according to embodiments herein;

[00018] Figure 5 is a magnified cross sectional side view of the second embodiment of the present tool depicted in Figure 4; and

[00019] Figure 6 is a cross section side view of a third embodiment of the present tool according to embodiments herein.

DESCRIPTION OF THE EMBODIMENTS

[00020] Embodiments herein relate to apparatus and methodology of amplifying or magnifying the transmission of fluid pressure into axial force for use in various applications where continuous, high-frequency axial force is desirable. Withoutlimitation, it is among the aspects of the present apparatus and methodologies to enable generate amplified movement, agitation or vibration of downhole drilling equipment. Such amplification may be generated by the present technology alone, or in combination with other downhole percussion-generating equipment. As such, the present apparatus and method may be operably configured to fluid-transmitting downhole drilling assemblies (e.g., drill string, coil tubing, casing string etc.) positioned within a borehole. It is understood that in other aspects the present technology may be also configured for use in hammer drilling applications, percussion motor applications, etc. The presentapparatus and methodologies is described below with references to the accompanying Figs. 1 – 6.

[00021] Having regard to Figs. 1 and 4, the present apparatus **10** comprises a tool body formed from at least two telescoping tubular elements **12**,**14** having inlet and outlet ends

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16,18, respectively, each tubular element forming central bores adapted to permit the passage of drilling fluid therethrough. Inlet and outlet ends **16**,**18** can include interior and exterior threading, as is known in the art, for operatively connecting the tool **10** with drill string, conventional percussion tools, or other downhole equipment as desired. For example, either inlet or outlet end **16**,**18** may comprise pin and box threading standard in the industry for operably connecting tool body **10** with known vibration tools (notshown), including tools capable of creating tunable pressure pulses. Either inlet or outlet end **16**,**18** may comprise standard pin and box threading for operatively connecting the tool body **10** with a shock sub **13** (e.g., a conventional shock absorber or vibration dampener). These connections allow the percussion tool (not shown), the present tool **10** and a conventional shock sub **13** to act as a single apparatus for imparting amplified axialloads.

[00022] As more clearly depicted in Figs. 2 and 5, first outer tubular element **12** (also referred to as the "housing") comprises a cylindrical wall forming a central bore **22** extending along a longitudinal axis ("A") between inlet end **16** and outlet end **18** downhole. Central bore **22** is operatively connected to receive volumes of pressurized fluid from the downhole percussion tool (not shown) being amplified. First tubularelement **12** can be of steel construction, or any other suitable material, and can be surface hardened for durability and abrasion resistance.

[00023] Second inner tubular element **14** (also referred to herein as the "piston") comprises a cylindrical wall forming a central bore **24**, fluidically connected with central bore **22**. Tubular piston **14** is configured to be telescopically disposed within outerelement 12, such that the two tubular elements **12**,**14** coaxially align to each have a central axis coincident with longitudinal axis "A". Tubular elements **12**,**14** are further operably connected to enable reciprocal extension and compression of the piston **14** during "firing" (e.g., opening and closing) of the tool **10**. It is contemplated that one or more additional pistons **14** (not shown) may be telescopically positioned within the tool **10**, further amplifying the loads imparted thereby. Second tubular element **14** can be of steel construction, or any other suitable material, and can be surface hardened for durability and abrasion resistance.

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[00024] Having regard to Fig. 3, in some embodiments, the present tool **10** comprises a first fluid chamber **20** for receiving fluid and operative to transmit fluid pressures topiston **14**, imparting movement thereof. It is to be understood that first fluid chamber **20** are adapted to receive varying volumes of fluid having varying fluid pressure. In some embodiments, first fluid chamber **20** may be positioned at or near inlet end **16**, and fluidically connected to piston bore **24**. In one embodiment, first fluid chamber **20** may be disposed between outer housing and inner piston elements **12**,**14**. First fluid chamber **20** may comprise a sealed cavity formed between the inner surface of the housing **12** and the outer surface of the piston **14**. In operation, pressurized fluid may flow into the tool **10** via inlet **16** and enter first fluid chamber **20**. Where sufficient fluid volume and/or hydraulic pressure within first fluid chamber **20** is achieved, forces generated induceaxial movement of piston **14** downwardly, firing the tool **10** (e.g., hydraulic fluid pressures converted to kinetic energy).

[00025] More specifically, without limitation, as fluid pressure (P_f) in fluid chamber **20** increases, the force imposed on surface areas A_1 and A_2 (up and down arrows, respectively) of chamber **20** cause piston **14** to telescope within housing 12. Downward axial movement of piston **14** compresses vibration-absorbing elements of shock tool **13**, converting the kinetic energy to stored energy. As $P_f$decreases within the tool **10**, vibration-absorbing elements reconfigure to achieve equilibrium, releasing stored energy as kinetic energy and causing the piston **14** to telescope back upwardly relative to the housing **12**. Both upward and downward movements of the piston **14** generate axial forces.

[00026] It is understood that the present tool **10** is configured to impart amplified axial loads upwards and downwards with high frequency (e.g., multiple times per second) over extended periods of time (e.g., approximately hundreds of hours, or preferably over approximately 200 hours). It is further understood that amplification of axial loads achieved by the present tool **10** may be sufficient to impart movement of weightydownhole equipment (e.g., at least approximately 350 meters of drill pipe). Without limitation, it is estimated that the present tool **10** may amplify the axial loads generated by known downhole percussion tools by approximately 65%.

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[00027] As would be understood, the axial force resulting from the present tool **10** is limited by the size of first chamber **20** which in turn is restricted by the diameter and overall size of the tool **10**. It is one aspect of the present apparatus and methodologies to increase the overall surface area within the at least one first chamber **20** that is acted upon by the pressurized fluid, thereby amplifying the axial forces attainable by the present tool **10**. As such, having further regard to Figs. 2 and 5, embodiments of the present tool **10** may be configured to provide at least one additional first fluid chamber **20***a*,**20***b*...**20***i*, the at least one additional first fluid chambers **20***a*,**20***b*,...**20***i* fluidically connected to each other via the piston bore **24** and at least one piston fluid port(s) **17** for cumulatively magnifying (i.e., at least doubling) the surface areas acted upon by pressurized fluid. Indeed, each additional first fluid chamber **20**,**20***a*, may provide additional surface areas A_1, A_2, A_3, A_4. It is understood that the overall axial force generated by the tool **10** is:

$$P_f = A_1 + A_2 + A_3 + A_4.$$

[00028] In order to prevent opposing forces (e.g., downward forces acting upon surface area A_2 vs. upward forces acting upon surface area A_3), and to ensure that P_f changes are cumulative and magnified, embodiments of the present tool **10** further comprise at least one second fluid chamber **26**. Second fluid chamber **26** may be a sealed fluid chamber having a predetermined and fixed volume of fluid at a fixed pressure.

[00029] Having regard to Figs. 2 and 5, according to embodiments herein, the at least one second fluid chamber **26** may also be disposed in between the at least two first fluid chambers **20**,**20***a*. More specifically, the second fluid chamber **26** may also be positioned between inner and outer tubular members **12**,**14**, such that the at least two first fluid chambers **20**,**20***a* are positioned thereabove and therebelow (e.g., movement of piston **14** due to P_f changes in first chambers **20**,**20***a* produces corresponding P_f changes in second chamber **26**). It is one aspect of the present apparatus and methodology that the second fluid chamber **26** be configured relative to the at least two first fluid chambers **20**,**20***a* so as to be capable of receiving and resisting opposed axial forces generated by increases in P_f(e.g., downward forces imposed on surface A_2 vs. upward forces imposed on surface A_3) in the at least two first fluid chambers **20**,**20***a*.

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[00030] In embodiments herein, each second fluid chamber **26** may comprise pressure compensation means for responding to P_f changes within the chamber **26**. More specifically, second fluid chamber **26** may be fluidically connected to the outside of the tool **10** via a plurality of radial fluid ports **30** extending through the sidewall of outer tubular element **12**. As the axial movement piston **14** compresses fluid in second fluid chamber **26**, P_f increases within second fluid chamber **26** and the fluid within chamber **26** (having fixed volume and pressure) is vented from the chamber **26** through fluid ports **30**. In order to prevent the loss of the vented fluid, pressure compensation means may further comprise a diaphragm **32**, the diaphragm encircling outer tubular member **12** and sealing fluid ports **30**. Diaphragm **32** may be comprised of any pressure-absorbent material capable of sealably capturing pressurized fluid venting through the fluid ports **30**. It is understood that diaphragm **32** further prevents contamination of the pressurized fluid within chamber 26 with fluids and debris outside the tool **10** (e.g., annular debris in the wellbore).

[00031] Embodiments herein further relate to methods of amplifying the transmission of fluid pressure into axial forces. In embodiments herein, the method may comprise providing the present tool **10** for use alone, or in combination with other downhole percussion or vibration-generating tools, wherein the present tool **10** may be utilized to amplify the vibrations. For example, without limitation, the present tool **10** may be utilized in combination with known percussive tools, such as those tools described in United States Patent No. 8,167,051, United States Patent Application No. 13/381,297 or PCT/CA2014/000701, particularly in extended reach drilling operations (imparting loads on hundreds of meters of pipe), or difficult drilling operations (e.g., soft/hard formations). It is an aspect of the present method that the tool **10** may be configured or tuned to provide high-frequency force (e.g., multiple "fires" per second) continuously or near- continuously for extended periods of time (e.g., up to hundreds of hours).

[00032] Embodiments herein further relate to apparatus and methodology of suppressing or dampening vibrations or axial forces generated by downhole equipment for use in various applications where reducing large and continuous axial forces is desirable. Without limitation, it is among the aspects of the present apparatus and methodologies to enable suppression of movement, agitation or vibration of downhole drilling equipment, such as

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pressures or “noise” generated by downhole drilling motors. As such, the present apparatus and method may be operably configured to fluid-transmitting downhole drillingassemblies (e.g., drill string, coil tubing, casing string etc.) positioned within a borehole, although it is understood that in other aspects the present technology may be also configured for use with Logging While Drilling (LWD) or Measurement While Drilling (MWD) tools, thereby improving the signal quality transmitted to the surface.

[00033] Having regard to Fig. 6, inlet and outlet ends **16**,**18** of the present tool **10** can include interior and exterior threading, as is known in the art, for operatively connecting the tool **10** with LWD or MWD tools (not shown), or other downhole equipment as desired. For example, either inlet or outlet end **16**,**18** may comprise pin and box threading standard in the industry for operably connecting tool body **10** with known downhole tubing or equipment. Central bores **22**,**24** are operatively connected to receive pressurizedfluid from the downhole dampening tool (not shown) magnifying the “noise-reducing” capacity of the dampening tool. As above, tubular elements **12**,**14** are configured to be telescopically disposed one within the other to enable reciprocal extension and compression of piston **14** within housing **12** during vibration dampening (e.g., absorption) of the tool **10**.

[00034] In operation, vibration of downhole equipment will cause compression of vibration-absorbing elements in the shock sub **13**. Compression of the dampening elements increases fluid pressure (P_f) in fluid chambers **20**,**20*a***, causing piston **14** to telescope upwardly within housing **12**, said $P_f$absorbed by pressure-absorption means of second fluid chamber **26**. Such operation may also serve absorb or reduce the pressure fluctuations or “noise” generated by drilling motors. It is contemplated that in such operations, the present tool **10** may configured to be used alone or in combination with downhole shock tools.

[00035] Without limitation, it would be understood by a person skilled in the art that in operation, the present tool **10** may be utilized to amplify the axial forces created by downhole percussion tools or, when configured to operate in reverse, to suppress or dampen the subsurface vibrations or “noise” created by downhole equipment.

[00036] Although a few embodiments have been shown and described, it will be

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appreciated by those skilled in the art that various changes and modifications might be made without departing from the scope of the invention. The terms and expressions used have been used as terms of description and not of limitation, and there is no intention in the use of such terms and expressions of excluding equivalents of the features shown and described or portions thereof, it being recognized that the invention is defined and limited only by the claims that follow.

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<u>WE **CLAIM:**</u>

1. An apparatus for receiving pressurized hydraulic fluid from at least one hydraulic fluid-transmitting downhole drilling tool and for amplifying changes in the hydraulic fluid pressures of the received fluid to generated amplified axial forces on the at least one downhole drilling tool, the apparatus being adapted to permit the passage of the hydraulic fluid therethrough, comprising:

    a tubular housing having a sidewall forming a central housingbore,
    a tubular piston, telescopically received within the housingbore, the piston having a sidewall forming a central piston bore, the piston bore being fluidically connected to the housing bore via at least one piston fluid port,
    at least two first hydraulic fluid chambers for receiving the hydraulic fluid, each of the at least two first hydraulic fluid chambers formed between the housing and piston sidewalls and directly fluidically connected via the piston bore and the at least one piston fluid port, wherein when the hydraulic fluid is received within the at least two first hydraulic fluid chambers, changes in the hydraulic fluid pressures within the at least two first fluid chambers are cumulative to impart amplified axial movement of the tubular piston relative to the tubular housing; and
    at least one second fluid chamber disposed in between the at least two first hydraulic fluid chambers, the at least one second fluid chamber being fluidly sealed from the at least two first hydraulic fluid chambers.

2. The apparatus of claim 1, wherein the at least one second fluid chamber comprises a fixed volume of fluid at a fixed pressure.

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3. The apparatus of claims 1 or 2, wherein the at least one second fluid chamber is formed between the tubular housing and the tubular piston.

4. The apparatus of any one of claims 1 – 3, wherein the at least one second fluid chamber further comprises a plurality of radial fluid ports disposed through the housing sidewallfor venting the fluid from the second fluid chamber through the housing sidewall.

5. The apparatus of claim 4, wherein the at least one second fluid chamber further comprises

   an annular membrane encircling the tubular housing for sealing the fluid ports and preventing the fluid from exiting the tool.

6. The apparatus of any one of claims 1 - 5, wherein the at least one second fluid chamber isoperative to resist opposed axial forces from the at least two first fluid chambers.

7. A method for amplifying the transmission of fluid pressure into axial forces, the method comprising:
   providing an amplification tool adapted to permit the passage of pressurized fluid therethrough, having:

   a tubular housing with a sidewall forming a central housingbore capable of receiving the fluid,

   a tubular piston, telescopically received within the housing bore, the piston having a sidewall forming a central piston bore, thepiston bore being fluidically connected to the housing bore via at least one piston fluid port, and

   at least two first fluid chambers, each of the at least two first fluid chambers directly fluidically connected via the piston bore and the at least one piston fluid port such that increases in fluid pressures within the at least two first fluid chambers accumulate to induce movement of the tubular piston relative to the tubular housing; and

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utilizing the amplification tool to generate axial forces.

8. The method of claim 7, wherein the amplification tool provided further comprises at least one second fluid chamber having pressure compensation means comprising a plurality of radial fluid ports disposed through the housing sidewall for venting the fluid from the second chamber through the housing sidewall.

9. The method of claim 8, wherein the pressure compensation means are operative to receive and resist opposed axial forces from the at least two first fluid chambers.

10. The method of any one of claims 7 – 9, wherein the method further comprises:

providing a percussion tool operatively connected to the amplification tool for generating an axial force, and

utilizing the amplification tool to amplify the axial force generated by the percussion tool.

11. An apparatus for receiving pressurized hydraulic fluid from at least one hydraulic fluid-transmitting downhole drilling tool, and for dampening changes in the hydraulic fluid pressures of the received fluids, the apparatus being adapted to permit the passage of fluid therethrough, comprising:

a tubular housing having a sidewall forming a central housing bore,

a tubular piston, telescopically received within the housing bore, the piston having a sidewall forming a central piston bore, the piston bore being fluidically connected to the housing bore via at least one piston fluid port,

at least two first hydraulic fluid chambers for receiving the hydraulic fluid, each of the at least two first fluid hydraulic fluid chambers formed between the housing and piston sidewalls and directly fluidically connected via the piston bore and the at least one piston fluid port, wherein when the hydraulic fluid is received within the at least two first hydraulic fluid chambers, changes in the hydraulic fluid pressures within the at least two first fluid chambers cumulatively absorb the fluid

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pressures by imparting movement of the tubular piston relative to the tubular housing; and

at least one second fluid chamber disposed in between the at least two first hydraulic fluid chambers, the at least one second fluid chamber being fluidly sealed from the at least two first hydraulic fluid chambers.

12. The apparatus of claim 11, wherein the at least one second fluid chamber comprises a fixed volume of fluid at a fixed pressure.

13. The apparatus of claim 11, wherein the at least one second fluid chamber is formed between the tubular housing and the tubular piston.

14. The apparatus of any one of claims 11-13, wherein the at least one second fluid chamber further comprises a plurality of radial fluid ports disposed through the housing sidewall for venting the fluid from the second chamber through the housing sidewall.

15. The apparatus of any one of claims 12 – 14, wherein the at least one second fluid chamber further comprises an annular membrane encircling the first tubular housing, for sealing the fluid ports and preventing fluid from exiting the tool.

16. The apparatus of any one of claims 12 – 15, wherein the at least one second fluid chamber is operative to resist opposed axial forces from the at least two first fluid chambers.

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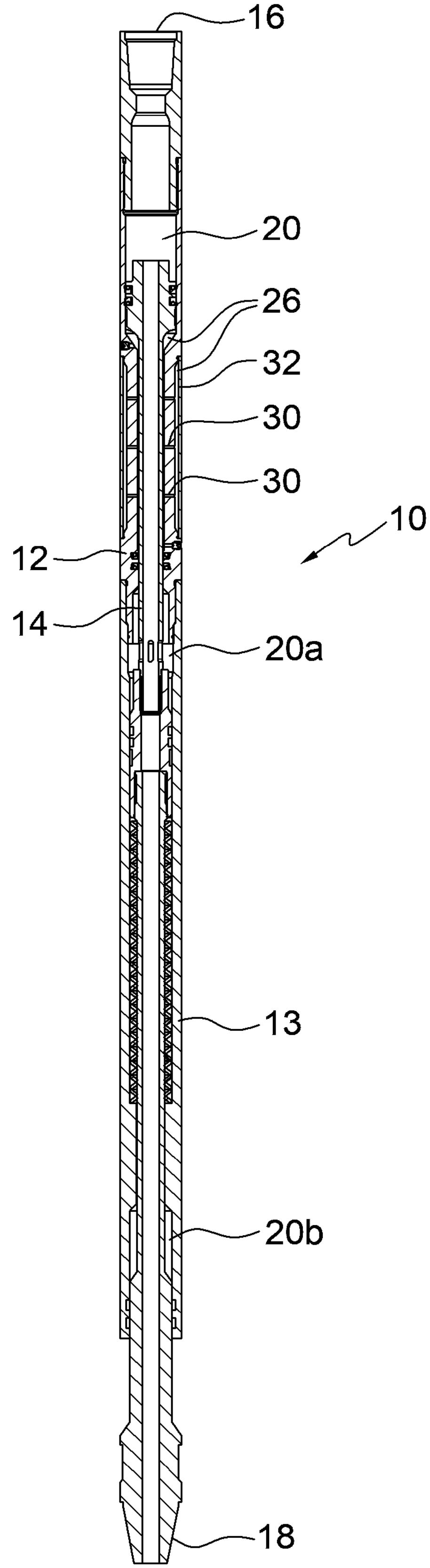

FIG. 1

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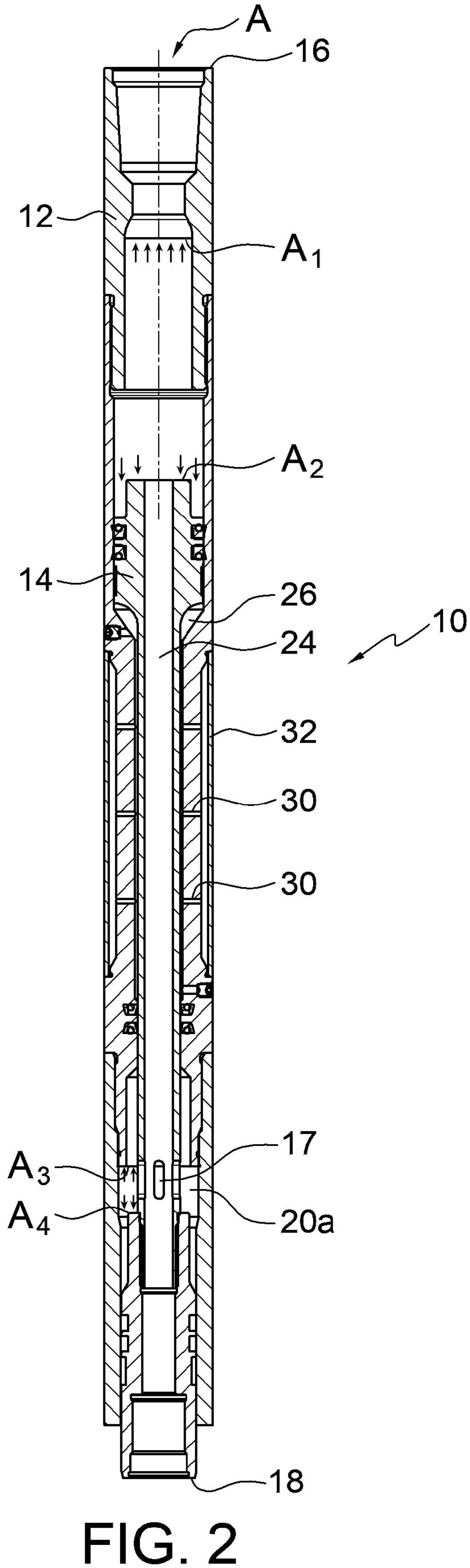

FIG. 2

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Replacement Sheet

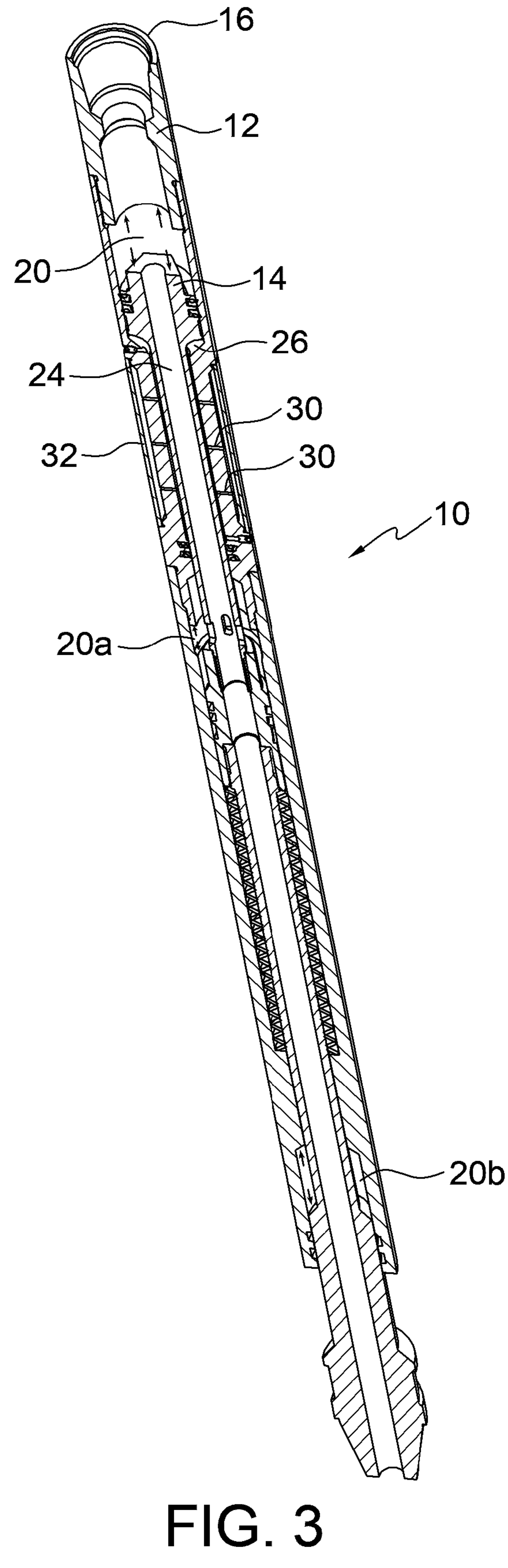


FIG. 3

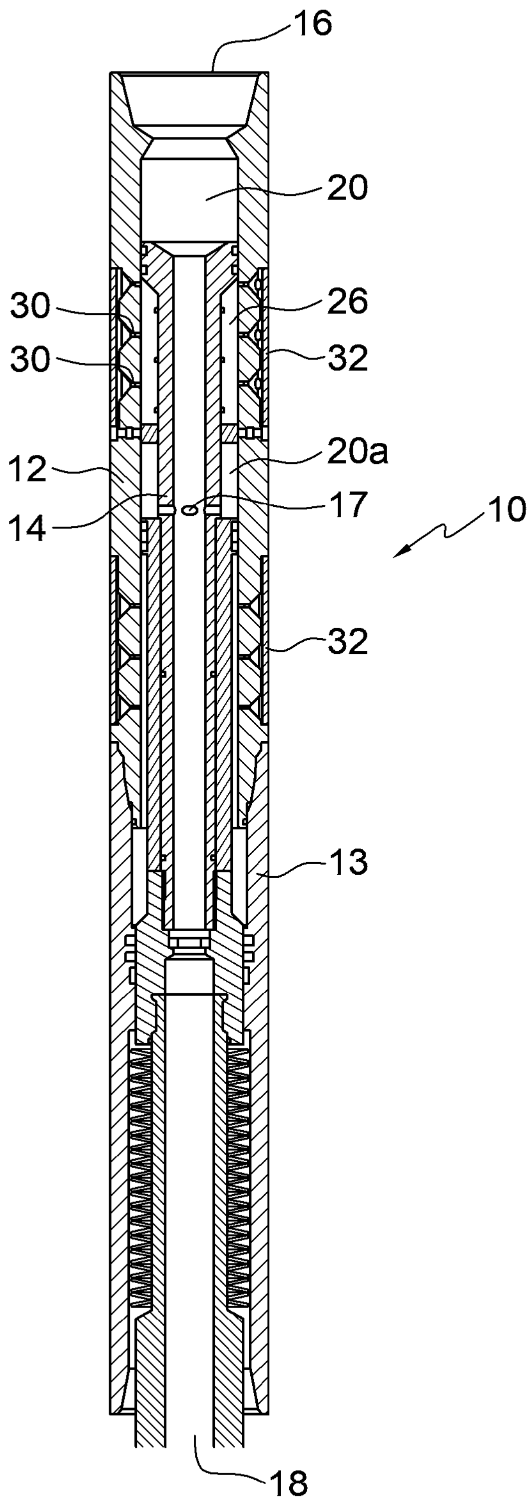


FIG. 4

Date Reçue/Date Received 2021-05-20

Replacement Sheet

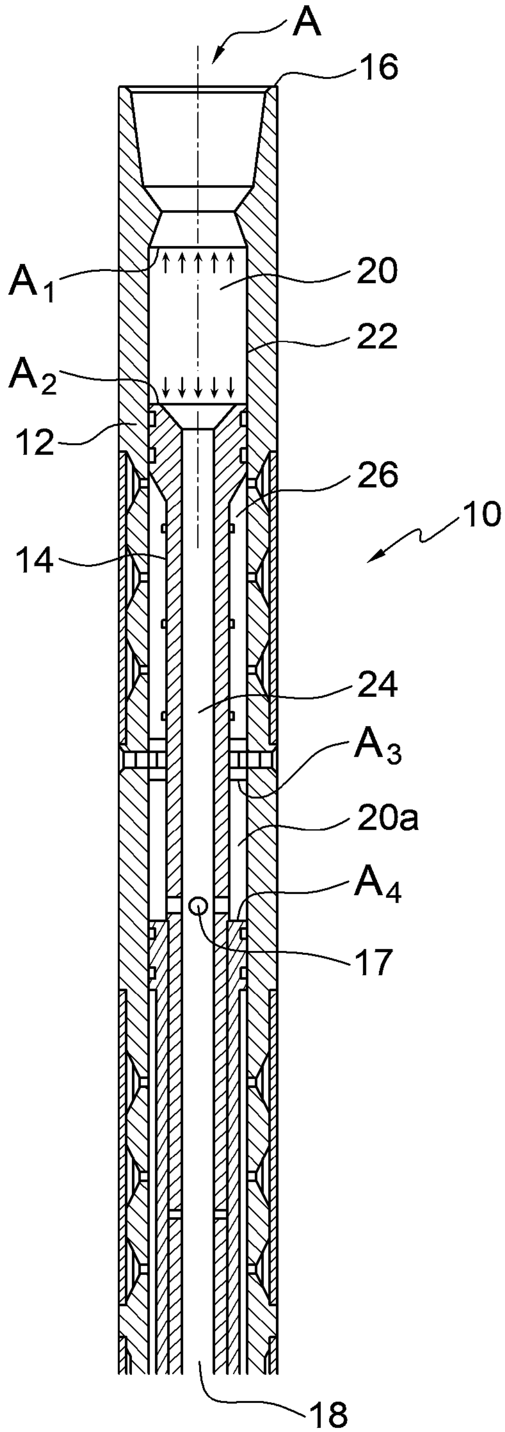


FIG. 5

Date Reçue/Date Received 2021-05-20

Replacement Sheet

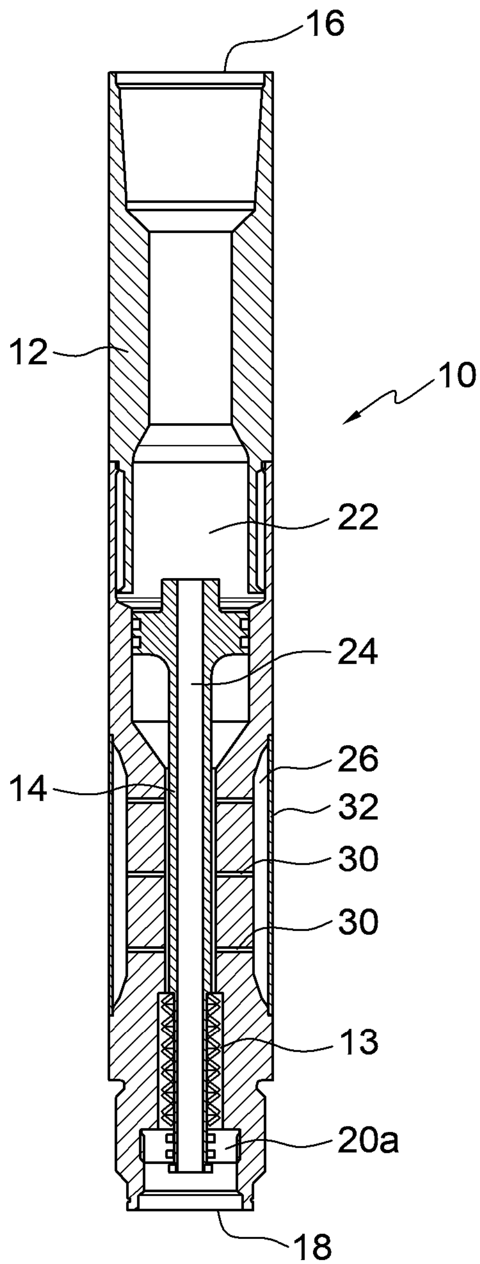
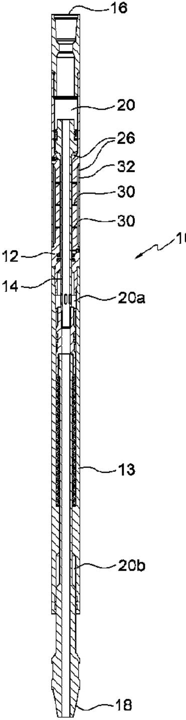


FIG. 6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