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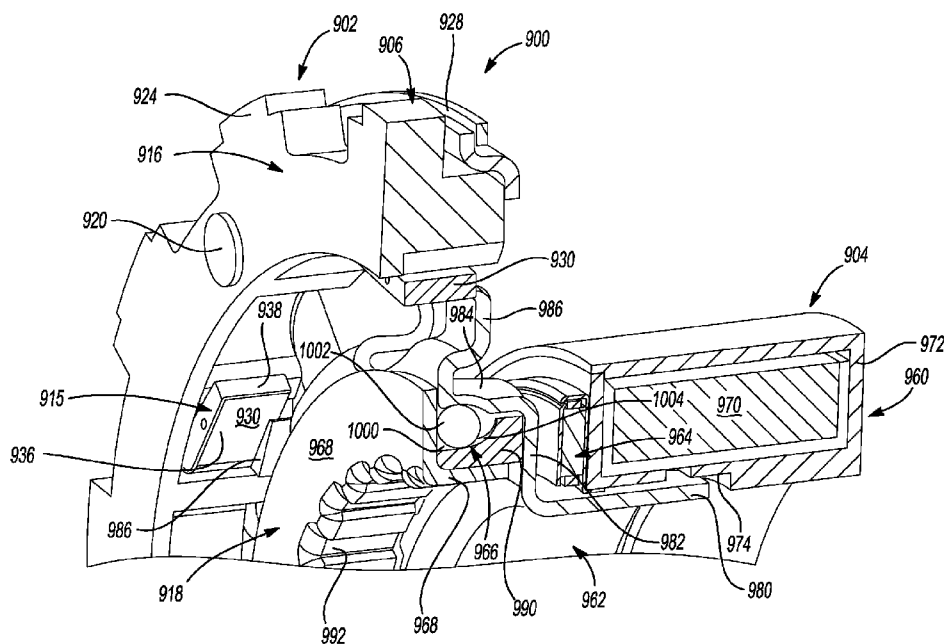


Fig-19

(57) Abstract: A controllable one-way clutch having a clutch module and a power-operated actuator module. The clutch module includes a first clutch component rotatably driven by an input member, a second clutch component rotatably driving an output member, struts disposed on the first clutch component for movement between a deployed position engaging ratchet teeth on the second clutch component and a non-deployed position disengaged from the ratchet teeth, and strut springs for biasing the struts toward their deployed position. The power-operated actuator module includes a coil unit, an armature moveable between an engaged position whereat the armature holds the struts in their non-deployed positions when the coil unit is powered-off and a released position where the armature releases the struts when the coil unit is powered-on, and a ballramp mechanism configured to move the armature along a helical path between its engaged and released positions.

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- *as to applicant's entitlement to apply for and be granted a patent (Rule 4.17(ii))*
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ROTATING ELECTRICALLY-CONTROLLED ONE-WAY CLUTCH (eOWC) WITH BALL CAM ACTUATION MECHANISM

CROSS-REFERENCE TO RELATED APPLICATIONS

[0001] This patent application claims the benefit and priority of U.S. Provisional Patent Application Serial No. 62/796,717, filed January 25, 2019, the entire disclosure of which is incorporated herein by reference.

FIELD OF THE INVENTION

[0002] The present disclosure is generally related to overrunning coupling devices and, more specifically to electrically-controlled one-way clutch (eOWC) devices having an electromagnetic actuator. The present disclosure is further related to use of such eOWC devices in motor vehicle driveline applications to provide a disconnect feature when equipped within electric drive axle and/or electric transaxles as well as when equipped within drive axle assemblies.

BACKGROUND OF THE INVENTION

[0003] This section provides background information related to the present disclosure which is not necessarily prior art.

[0004] Clutches and/or brakes are used in a variety of motor vehicle applications to establish a torque-transmitting drive connection between an input component and an output component for supplying motive power (i.e., drive torque) from a powertrain to a driveline. One type of brake or clutch widely used is an overrunning coupling device, commonly referred to as a one-way clutch (OWC), which overruns when one of its races (in radial coupling configuration)

or one of its drive plates (in axial coupling configurations) rotates in a first (i.e., freewheel) direction relative to the other race or drive plate, and engages or locks in a second (i.e., lockup) direction. Such conventional one-way clutches provide no independent control over their modes of operation, that is to say whether they lockup or freewheel in both directions and are commonly referred to as passive one-way clutches. Thus, basic one-way clutches provide a “Locked” mode in one rotary direction and a “Freewheel” mode in the opposite direction based on the direction that the drive torque is being applied to the input race or drive plate.

[0005] There are however, requirements where a “controllable” overrunning coupling device, also commonly referred to as a selectable one-way clutch, can be selectively controlled to provide additional functional modes of operation. Specifically, a selectable one-way clutch may further be capable of providing a Freewheel mode in both rotary directions until a command signal (i.e., from a vehicle controller) causes a power-operated actuator to shift the coupling device into its Locked mode. Thus, a selectable one-way clutch may be capable of providing a drive connection between an input member and an output member in one or both rotational directions and it may also be operable to freewheel in one or both directions. It is also known to integrate a passive one-way clutch and a selectable one-way clutch into a combined coupling device, commonly referred to as a bi-directional clutch.

[0006] In addition to traditional motor vehicles, significant development activity is currently directed to electric transaxles and electric drive axles which are driven by an electric motor for providing the motor vehicle with an electric drive mode. In such devices, a mechanical coupling device is employed to selectively disconnect the electric motor from a geartrain (i.e., multi-speed transmission, final drive unit, planetary gearset, etc.) to prevent unnecessary drag when the electric drive mode is not selected. Conversely, the mechanical

coupling device also functions to drivingly connect the electric motor to the geartrain when the electric drive mode is selected. Currently, these mechanical coupling devices, commonly referred to as “disconnect” clutches, are of the positive-engagement dog-clutch configuration and are only capable of providing ON/OFF functionality. Such dog-type disconnect clutches typically have high backlash angles, require very high release forces to disengage, and do not allow a ratcheting (i.e. one-way freewheeling) behavior to accommodate a unidirectional overrun condition.

[0007] In view of the above, a need exists to continue development of new and improved overrunning coupling devices that advance the art and provide enhanced functionality. Specifically, a need exists to develop alternative disconnect clutches providing reduced backlash tooth engagement and which are capable of additional modes of operation, such as a freewheeling mode.

SUMMARY

[0008] The following listing of various aspects, features and embodiments of the present disclosure is not intended to be an exhaustive and comprehensive summation of all the inventive concepts disclosed herein and protected by the claims of this application

[0009] It is an aspect of the present disclosure to provide a controllable one-way clutch configured to provide a disconnect function when installed in a motor vehicle driveline.

[0010] It is a related aspect of the present disclosure to configure the controllable one-way clutch as an axle disconnect clutch to provide the disconnect function in association with a drive axle assembly of AWD/4WD vehicles.

[0011] It is an alternative aspect of the present disclosure to configure the controllable one-way clutch as a motor disconnect clutch in an electric transaxle or an electric drive axle to provide the disconnect function for selectively disconnecting an electric motor with respect to a geartrain.

[0011] It is yet another related aspect of the present disclosure to configure the controllable one-way clutch as an electrically-controlled one-way clutch (eOWC) having a clutch module and a power-operated actuator. The clutch module may include rotary first and second clutch components, a plurality of struts supported on the first clutch component for movement between a first or “deployed” position engaged with ratchet teeth formed on the second clutch component and a second or “non-deployed” position displaced from engagement with the ratchet teeth, and strut springs normally biasing the struts toward their deployed positions. The power-operated actuator being operable in a first or “power-off” condition to mechanically engage the struts and hold the struts in their non-deployed positions and in a second or “power-on” condition to disengage the struts and allow the strut springs to locate the struts in their deployed positions.

[0012] It is a still further aspect of the present disclosure to configure the power-operated actuator as an electromagnetic actuator assembly having a stationary coil unit, an armature moveable relative to the coil unit and the struts between a first or “engaged” position whereat the armature engages and holds the struts in their non-deployed positions and a second or “released” position whereat the armature is displaced from engagement with the struts, an armature biasing arrangement configured to normally bias the armature toward its engaged position, and a ballramp mechanism disposed between the clutch module and the armature. Energization of the coil unit establishes the power-on condition and causes the armature to move

from its engaged position to its released position in opposition to the biasing force exerted on the armature by the armature biasing arrangement. Subsequent de-energization of the coil unit establishes the power-off condition and permits the armature biasing arrangement to drive the armature back to its engaged position. A “Freewheel” mode is established in the power-off condition to provide the disconnect function. A “Latched” mode is established in the power-on condition to permit torque transfer in a first relative rotary direction and ratcheting in a second relative rotary direction.

[0013] It is yet another related aspect of the present disclosure to configure the ballramp mechanism to cause the armature to travel along a helical path between its engaged and released position.

[0014] These and other features and advantages of the present invention will become more readily appreciated when considered in connection with the following detailed description and appending drawings.

BRIEF DESCRIPTION OF THE DRAWINGS

[0015] The drawings described herein are for illustrative purposes only of selected embodiments and are not intended to limit the scope of the present disclosure. The inventive concepts associated with the present disclosure will be more readily understood by reference to the following description in combination with the accompanying drawings wherein:

[0016] FIG. 1 is an exploded isometric view of a bi-directional clutch assembly configured to include a passive one-way clutch having passive struts, and a controllable one-way clutch having a pair of electromagnetic actuators and a pair of active strut assemblies;

[0017] FIG. 2 is a front view of a lead frame and an integrated safety switch for controlling the electromagnetic actuators of FIG. 1;

[0018] FIG. 3 is an assembled partial isometric view of the clutch assembly of FIG. 1 showing an arrangement for installing the electromagnetic actuators in an axially directed configuration;

[0019] FIGS. 3A and 3B are cross-sectional side views of the electromagnetic actuator of FIG. 3 with an active strut of the active strut assembly shown in deployed and non-deployed positions, respectively, in response to respective energized and de-energized states of the electromagnetic actuator;

[0020] FIGS. 4A and 4B are similar views to FIGS. 3A and 3B but show an electromagnetic actuator constructed in accordance with another aspect of the present disclosure;

[0021] FIG. 5 is an exploded isometric view of a bi-directional clutch assembly configured to include a modular active strut arrangement for a controllable one-way clutch in accordance with another aspect of the present disclosure;

[0022] FIG. 6 is a cross-sectional side view of the bi-directional clutch assembly shown in FIG. 5;

[0023] FIG. 7 is a sectional view of yet another embodiment of a bi-directional clutch assembly constructed in accordance with another aspect of the present disclosure;

[0024] FIG. 8 is an enlarged partial view taken from FIG. 7 and illustrating the configuration of one of the passive strut assemblies in greater detail;

[0025] FIG. 9 is another enlarged partial view taken from FIG. 7 and illustrating the configuration of one of the active strut assemblies in greater detail;

[0026] FIG. 10 is a sectional view of the bi-directional clutch assembly of FIG. 7 which better illustrates many of the components associated with the active strut assembly and an electromagnetic actuator assembly;

[0027] FIG. 11 is similar to FIG. 10 and now illustrates a magnetic flux path generated in response to energization of a coil unit and corresponding axial movement of an armature ring associated with the electromagnetic actuator assembly resulting from the magnetic flux path;

[0028] FIG. 12 is a partial isometric view showing the armature ring located in a non-actuated position with its strut engagement feature displaced from engagement with an active strut associated with the active strut assembly, whereby the active strut is maintained in its non-deployed position when the coil unit is in a non-energized state;

[0029] FIG. 13 is another partial isometric view, similar to FIG. 12, but now showing movement of the armature ring toward an actuated position for causing its strut engagement feature to engage the active strut and forcibly pivot the active strut from its non-deployed position toward a deployed position in response to the coil unit being shifted into an energized state;

[0030] FIG. 14 is a plan view of FIG. 13;

[0031] FIG. 15 is a schematic view of the bi-directional clutch assembly of FIGS. 7-14 shown installed between a rotary input member and a rotary output member for providing a freewheeling disconnect function;

[0032] FIG. 16 is a schematic view of the disconnect-type bi-directional clutch assembly of FIGS. 7-14 now installed within an electric drive axle (EDA) assembly;

[0033] FIG. 17 is another schematic view of the disconnect-type bi-directional clutch assembly of FIGS. 7-14 now installed within another version of an EDA assembly;

[0034] FIG. 18 is a sectional isometric view of a controllable one-way clutch constructed in accordance with yet another aspect of the present disclosure;

[0035] FIG. 19 is another isometric view showing the controllable one-way clutch of FIG. 18 to include a clutch module having a plurality of passive strut assemblies, and an electromagnetic actuator assembly having a coil unit, a moveable armature, and a ballramp mechanism;

[0036] FIG. 20 is an end view of the controllable one-way clutch shown in FIG. 19;

[0037] FIGS. 21 and 22 are additional views providing more clarity to the components associated with the ballramp mechanism;

[0038] FIGS. 23 through 25 are views providing additional clarity to the components of an armature biasing arrangement associated with the electromagnetic actuator assembly;

[0039] FIG. 26 is a partial isometric view illustrating a lost-motion type of travel-limiting arrangement disposed between the armature and the ballramp mechanism;

[0040] FIGS. 27 through 30 illustrate the controllable one-way clutch of FIGS. 18-26 operating in a first mode with the coil unit in a power-off condition and the armature located in an engaged position for mechanically engaging and holding the struts in a non-deployed position so as to establish a disconnect function; and

[0041] FIGS. 31 and 32 illustrate the controllable one-way clutch of FIGS. 18-26 operating in a second mode with the coil unit in a power-on condition for moving the armature

into a released position displaced from the struts for permitting movement of the struts into a deployed position engaged with ratchet teeth.

DESCRIPTION OF THE ENABLING EMBODIMENTS

[0042] Example embodiments will now be described more fully with reference to the accompanying drawings. In general, each embodiment is directed to an overrunning coupling device (i.e. brake and/or clutch) having at least a controllable one-way locking device including a moveable locking component (i.e. sprag, strut, etc.) that is controlled, at least in part, via an electromagnetic actuator. Thus, the overrunning coupling device transmits torque mechanically but is shifted between operating modes via an electrical actuation system. It is to be recognized the example embodiments only are provided so that this disclosure will be thorough, and will fully convey the scope, which is ultimately defined by the claims, to those who are skilled in the art. Numerous specific details are set forth, such as examples of specific components, devices, and methods, to provide a thorough understanding of embodiments of the present disclosure. It will be apparent to those skilled in the art that certain specific details need not be employed, that example embodiments may be embodied in many different forms, and that neither should be construed to limit the scope of the disclosure or the claims. In some example embodiments, well-understood processes, well-understood device structures, and well-understood technologies are not described in detail.

[0043] Referring to FIGS. 1, 3 and 3A-3B, wherein like numerals indicate corresponding parts throughout the several views, unless otherwise stated, a bi-directional clutch assembly **20** is generally shown. As will be detailed in further detail below, bi-directional clutch assembly **20** generally includes a stationary outer race, a rotatable inner race,

a passive one-way clutch having a plurality of passive struts, and a controllable one-way clutch having at least one active strut assembly and an electromagnetic actuator associated with each active strut assembly. The clutch assembly **20** includes an outer housing presenting an outer race **22** that extends annularly about an axis **A**. It should be recognized the outer housing/outer race **22** can be constructed as a single piece of material or as separate pieces of material and subsequently fixed to one another, as will be appreciated by one skilled in the art. The outer race **22** includes an outer ring segment **24** and an inner ring segment **26** that are spaced radially from one another and interconnected via a radial web segment **27**. The outer ring segment **24** presents a plurality of outer lugs **28** that extend radially outwardly from an outer surface **29** of the outer ring segment **24** for mating with a first component. The first component can be a stationary component (such as a housing of a transmission) or a rotary component (such as a shaft). The outer ring segment **24** further presents at least one, and shown as a pair of coil assembly support protrusions, referred to hereafter simply as protrusions **30**, that extend radially outwardly from the outer surface **29** of the outer ring segment **24**. Each of the protrusions **30** forms a radially extending actuator pocket **32** and a strut pocket **33**. The respective pockets **32**, **33** are shown as being open along one side surface **35**, and closed along an opposite side surface **37** by a wall of the protrusion **30**. It should be appreciated that more or fewer protrusions **30** could be provided by the outer ring segment **24**. The inner ring segment **26** presents a plurality of inner ramp surfaces, hereinafter referred to as inner ratchet teeth **34**, that extend radially inwardly and are evenly distributed about the axis **A**.

[0044] The clutch assembly **20** further includes an annular inner race **36** that also extends annularly about the axis **A**. The inner race **36** has an outer rim **38** and an inner rim **40** that are spaced radially from one another by a radial web segment **41**. Upon assembly, the

outer rim **38** is disposed radially between the outer and inner ring segments **24**, **26** of the outer race **22**, and the inner rim **40** is disposed radially inwardly from the inner ring segment **26** of the outer race **22**. The inner rim **40** of the inner race **36** presents a plurality of inner lugs **42** that extend radially inwardly for mating with a second component (typically a rotary component). Commonly, lugs **42** interconnect a shaft or clutch plates for conjoint rotation with inner race **36**. Further, the outer rim **38** of the inner race **36** presents a plurality of outer ramp surfaces, hereinafter referred to as outer ratchet teeth **44**, that extend radially outwardly and are evenly distributed about the axis **A**.

[0045] The passive one-way clutch includes a plurality of locking elements, also referred to as passive struts **46**, which are pivotally supported in strut apertures formed in the inner race **36** for pivotal movement between a locking position and an unlocking position. In the locking position, the passive struts **46** engage the inner ratchet teeth **34** of the outer race **22** for connecting the outer and inner races **22**, **36** to one another during counter-clockwise rotation of the inner race **36** relative to the outer race **22**. Therefore, engagement by one or more of the passive struts **46** prevents relative displacement of the outer and inner races **22**, **36** in the counter-clockwise direction. However, the passive struts **46** still allow relative displacement between the outer and inner races **22**, **36**, i.e., overrun, in the clockwise direction when located in the locked position since they “ratchet” over the ramped profile of the inner ratchet teeth **34**. In the unlocked position, the passive struts **46** are radially spaced from the inner ratchet teeth **34** of the outer race **22**, therefore also allowing counter-clockwise rotation of the inner race **36** relative to the outer race **22**.

[0046] In association with the controllable one-way clutch, an active strut assembly **48**, a portion of which is received by each of the strut pockets **33** in the outer ring

segment **24**. Each of the active strut assemblies **48** includes an active strut **50** that is selectively pivotal between a deployed and a non-deployed position. In the deployed position (FIG. 3A), a portion of the active strut **50** lockingly engages the outer ratchet teeth **44** of the inner race **36**, thereby locking the outer and inner races **22, 36** to one another during clockwise movement of the inner race **36** relative to the outer race **22**. However, the active strut **50** still allows relative displacement between the outer and inner races **22, 36**, i.e., overrun, in the counter-clockwise direction. In the non-deployed position (FIG. 3B), the active strut **50** is radially spaced from the outer ratchet teeth **44**, allowing the inner and outer races **22, 36** to rotate relative to one another. Furthermore, each of the active strut assemblies **48** includes an armature **60** that is disposed adjacent to, and in operable communication with, the active strut **50** for providing the selective pivotal movement of the active strut **50**.

[0047] The controllable one-way clutch also includes an electromagnetic actuator **51**, which is associated in operable communication with each active strut **50**. Each electromagnetic actuator **51** includes a coil assembly **52**. The coil assembly **52** is mounted in the actuator pocket **32** and is radially spaced from the active strut **50** and armature **60**. The coil assembly **52** includes an elongate pole piece or core **54** of a paramagnetic or ferromagnetic, magnetically permeable material, shown by way of example and without limitation as being generally cylindrical, a bobbin **56** disposed about the core **54**, and at least one coil **58** spirally wrapped about the bobbin **56**. Furthermore, the armature **60** is a magnetic component disposed between the active strut **50** and the core **54** / coil **58**. The armature **60** pivots toward the core **54**, due to magnetic attraction caused via energization of the coil assembly **52**, and thus providing the pivotal movement of the active strut **50** in response to energization of the coil **58** via electric current.

[0048] More specifically, when electric voltage and/or current are applied to the coil 58, the coil 58 becomes an electromagnet producing an electric field (or magnetic flux). The flux flows outwards in all directions and transfers through the small preset air gap **G** (FIG. 3B) between the armature 60 and core 54, which extends through a central through passage of the bobbin 56 of the coil assembly 52. The core 54 becomes selectively magnetized by the energized coil 58, thereby attracting and pivoting the magnetically attracted metallic armature 60 towards the core 54. The resulting pivoting motion of the armature 60 forces the active strut 50 to mechanically deploy radially inwardly due to the linkage between the active strut 50 and the armature 60. On deployment, the active strut 50 moves from its non-deployed position (FIG. 3B) to its deployed position (FIG. 3A) where it locates itself against one of the outer ratchet teeth 44 of the inner race 36, effectively locking the inner race 36 from rotating clockwise relative to the outer race 22. Disengagement occurs as voltage and/or current is removed from the coil assembly 52, wherein the core 54 is selectively and suddenly demagnetized, thus releasing the armature 60 from being magnetically attracted to the core 54 of the coil assembly 52. A strut biasing spring 61 is positioned within the strut pocket 33 between an end portion of the active strut 50 and the outer race 22, causing the active strut 50 to pivot back to its non-deployed position upon de-energization of the coil assembly 52 and demagnetification of the core 54.

[0049] It should be appreciated that the arrangement of the armature 60, active strut 50, and coil assembly 52 can act to apply a locking force in a radial direction (as shown in FIG. 1) or an axial direction, depending on the layout and/or requirements of the clutch assembly 20. Radial stacked clutch assembly 20 designs offer packaging advantages over their axial counterparts in situations where axial space is tight, e.g., in automatic transmissions.

Further, radially applied clutches transmit driving torque directly outwards to be grounded against the transmission housing without the fear of forces being directed axially which could cause problems for the sizing of other system components to compensate for axial force.

[0050] As shown in FIGS. 1 and 2, a lead frame **62** is attached to each of the electromagnetic actuators **51** for electrically connecting the coils **58** to one another for simultaneously energizing the coils **58**. It should be appreciated that the lead frame **62** could connect any number of coils **58**. A printed circuit board (PCB) **64** is attached to the lead frame **62** for selectively controlling the energization of the coils **58**. The PCB **64** is disposed radially and axially adjacent to one of the coils **58**. The lead frame **62** further includes at least one power output contact **66** that is disposed radially and axially adjacent to each of the coils **58** for electrically connecting to the coils **58** to provide power to the coils **58**. Any number of power contacts **66** could be utilized to power any number of coils **58**. Any suitable connection can be utilized to connect the power output contact **66** and the coils **58**. Furthermore, at least one wire (not shown) extends between the circuit board **64** and each of the power output contacts **66** for electrically connecting the circuit board **64** and the power output contacts **66**. The lead frame **62** also includes connection interface **68** for establishing an electrical connection between the printed circuit board **64** and an external control module, such as an electronic control unit (ECV) for transmitting data to the PCB **64** and to power the PCB **64**. Additionally, the lead frame **62** includes a plastic encapsulation or casing **70** that is disposed about the PCB **64** and the electronics for protecting the PCB **64** and the electronics for allowing the lead frame **62** to be submerged in automatic transmission fluid (ATF) and operate in -40C to +140C temperatures. It should be appreciated that the aforementioned configuration of the lead frame **62** and associated

components provide a low-cost, modular solution that provides for a more simplified manufacturing process and assembly.

[0051] Referring to FIGS. 3, 3A and 3B, an example embodiment of the protrusions 30, shows the open side surface 35 defines the actuator pocket 32 which is configured for axially receiving the coil assembly 52. Furthermore, a radially outwardly facing outer wall 72 of the protrusion 30 provides a through passage, also referred to simply as orifice 74, that extends radially inwardly into the actuator pocket 32. In this embodiment, coil 58 and bobbin 56 are axially disposed and press fit into the actuator pocket 32 through the open side surface 35. Once in position in the actuator pocket 32, which extends radially inwardly from the outer wall 72, the core 54 is pressed radially inwardly through the orifice 74 and through a central through passage of the bobbin 56 until it reaches a predetermined, preset location which sets the magnetic gap G between an armature 60 and the free end of the core 54. The core 54 attains a press fit against an inner surface 75 that defines and bounds the through passage of the bobbin 56, thereby being fixed against movement therein. The core 54 can further be press fit in the through passage of the bobbin 56, or configured in a clearance fit therein. FIG. 3A shows an active strut 50 in the deployed position as a result of energization of the coil assembly 52 and FIG. 3B shows the active strut 50 in the non-deployed position as a result of de-energizing the coil assembly 52.

[0052] Referring to FIGS. 5 and 6, wherein like numerals indicate corresponding parts throughout the several views, offset by a factor of 400, another embodiment of a clutch assembly 420 is generally shown. The clutch assembly 420 includes an outer race 422 that extends annularly about an axis A. The outer race 422 includes an outer ring 424 that presents a plurality of outer lugs 428 that extend radially outwardly for mating with a first component. The

first component can be a stationary component (such as a housing of a transmission) or a rotary component (such as a shaft). The outer race **422** further has an axially facing web or face **427** that has an annular shape that extends radially inwardly from the outer ring **424**. A plurality of passive struts **446** are pivotally connected to the axial face **427**. A biasing spring (not shown) engages each of the passive struts **446** for biasing the passive struts **446** in a locked position toward an inner race **436**.

[0053] The inner race **436** extends annularly about the axis **A**. The inner race **436** has an outside rim or band **438** and an inside rim or band **440** that are spaced radially from one another on opposing sides of the passive struts **446**. The inside band **440** of the inner race **436** presents a plurality of inner lugs **442** that extend radially inwardly therefrom for mating with a second component (typically a rotary component). The inside band **440** of the inner race **436** further presents a plurality of passive teeth **434** that extend radially outwardly therefrom for being engaged by the passive struts **446** for locking the inner and outer races **436**, **422** to one another in response to counter-clockwise rotation of the inner race **436** relative to the outer race **422**. The outside band **438** of the inner race **436** presents a plurality of active teeth **444** that extend radially outwardly therefrom and are evenly distributed about the axis **A**.

[0054] A plurality of active strut assemblies **448** are axially connected to the outer race **422**. Each of the active strut assemblies **448** includes a generally arc shaped protrusion **430**, wherein the protrusion **430** is constructed as separate piece of material from the outer race **422**. Each protrusion **430** includes a base **82** and a pair of circumferentially extending flanges **84** that extend from the base **82** on opposing sides of the base **82**. A fastener **85**, e.g., a bolt, extends axially through a through opening in each of the flanges **84** and is fastened to the outer race **422** for securing the active strut assemblies **448** to the outer race **422**. The active strut

assemblies **448** are arranged in circumferential alignment with one another about the axis **A**, as desired.

[0055] An actuator pocket **432** extends axially into the base **82** of each of the active strut assemblies **448**. A coil assembly **452** is disposed in each of the actuator pockets **432**. The coil assembly **452** includes a core **454** of a magnetically permeable material, a bobbin **456** configured for receipt about the core **454**, and a coil **458** wrapped about the bobbin **456**. It should be appreciated that the bobbins **456** and coils **458** of the coil assemblies **452** can advantageously be easily fitted into their respective pockets **432** for easy installation.

[0056] Each of the active strut assemblies **448** includes an active strut **450** that is selectively pivotal between a deployed and a non-deployed position, as discussed above. In the deployed position, the active struts **450** engage the active teeth **444** of the inner race **436**, therefore locking the outer and inner races **422**, **436** to one another during clockwise movement of the inner race **436** relative to the outer race **422**. However, the active struts **450** allow relative displacement, i.e., overrun, in the counter-clockwise direction. In the non-deployed position, the active struts **450** are radially spaced from the active teeth **444**, allowing the outer and inner races **422**, **436** to rotate relative to one another.

[0057] The plurality of passive struts **446** are pivotal between a locking position and an unlocking position. In the locking position, the passive struts **446** engage the passive teeth **434** of the outer race **422** for connecting the outer and inner races **422**, **436** to one another during counter-clockwise rotation of the inner race **436** relative to the outer race **422**. Therefore, engagement by the passive struts **446** prevents relative displacement of the outer and inner races **422**, **436** in the counter-clockwise direction, however, the passive struts **446** allow relative displacement, i.e., overrun, in the clockwise direction. In the unlocking position, the

passive struts **446** are radially spaced from the passive teeth **434** of the outer race **422**, thereby allowing counter-clockwise rotation of the inner race **436** relative to the outer race **422**.

[0058] Each of the active strut assemblies **448** further includes an armature **460** disposed between the active strut **450** and the core **454** for providing the pivotal movement of the active strut **450** in response to energization of the coil **458**. A lead frame **462**, such as discussed above with regard to FIGS. 1 and 2, electrically connects the coils **458** to one another for energizing the coils **458** to actuate and pivot the active struts **450** to their engaged, locked positions.

[0059] Accordingly, it should be appreciated that the modular configuration of the active strut assemblies/coil assemblies **448**, **452** allows the active strut assemblies/coil assemblies **448**, **452** to be manufactured separately from the rest of the clutch assembly **420**. Further, it should be appreciated that any number of the active strut assemblies/coil assemblies **448**, **452** could be installed on any given clutch assembly **420** as needed to provide a needed amount of torque. Additionally, it should be appreciated that the modular active strut assemblies as described herein could be utilized on various other clutch assembly configurations.

[0060] Each of the bi-directional clutch assemblies disclosed in reference to FIGS. 1-6 are generally configured to include a stationary outer race and a rotary inner race with the passive and active struts arranged to establish a locked (i.e. “braked”) connection therebetween. Obviously, these bi-directional clutch assemblies could also be configured for use with rotary inner and outer races to establish “rotary” connections therebetween as well. In this regard, the present disclosure also contemplates alternative configurations of a bi-directional clutch assembly comprising: a rotary outer race; a rotary inner race; a passive one-

way clutch operable to normally establish a reasonable torque-transferring connection between the inner and outer races; and a controllable one-way clutch selectively actuated by an electromagnetic actuator to establish other torque-transferring connections between the inner and outer races. Particular applications of this type of bi-directional clutch assembly may include, without limitations, axle disconnect clutches in 4WD/AWD drivelines as well as motor disconnect clutches in electric transaxles and electric drive axles. Accordingly, the following embodiment illustrated in FIGS. 7-14 is directed to this type of electric disconnect (E-Disconnect) clutch assembly and provide enhancements with respect to traditional dog-type positive engagement clutches now used in conventional arrangements.

[0061] By way of background, there is a recognized need in hybrid vehicle applications to employ a mechanical coupling device to selectively connect and disconnect an electric motor with respect to a geartrain within an E-drive assembly (i.e. electric transaxle, electric drive axle, etc.). The mechanical coupling devices must be operable to selectively couple the rotary output of the electric motor to the geartrain (or other portions of the driveline) when needed to establish an electric drive mode, and to uncouple the rotary output of the electric motor from the geartrain when the electric drive mode is no longer required in order to prevent unnecessary drag. As noted above, conventionally a positive-engagement dog-type clutch is used in this vehicular application to establish the coupled/uncoupled (i.e. ON/OFF) operative states, typically via movement of a sliding lock sleeve. Unfortunately, such dog clutches require high backlash angles, require high release forces to disengage, and do not allow a ratcheting-type behavior (i.e. “freewheeling” in one direction) such as in the event of an overspeed condition when the geartrain rotates faster than the electric motor shaft.

[0062] To address these and other shortcomings, the present disclosure provides a bi-directional clutch assembly **600**. In general, bi-directional clutch assembly **600** includes a rotary inner race **602**, a rotary outer race **604** defining sets of passive strut pockets **606** and a set of active strut pockets **608**, a coil assembly **610** having a stationary (non-rotating) bobbin **612** supporting an annular coil unit **614** and an armature ring **616** which is fixed for rotation with outer race **604** and yet is axially moveable relative to coil unit **614**, a set of passive strut assemblies **618** each having a passive strut **626** pivotably supported in corresponding passive strut pocket **606**, and a set of active strut assemblies **620** each having an active strut **630** pivotably supported in corresponding active strut pocket **608**. FIG. 11 illustrates a non-limiting embodiment of bi-directional clutch assembly **600** with armature ring **616** removed. Inner race **602** has an outer surface defining ratchet teeth **622** each having a first strut engagement surface **624** configured to locking engage an engagement portion of passive struts **626**, and a second strut engagement surface **628** configured to selectively locking engage an engagement portion of active struts **630**. Inner race **602** can be a separate component or integrated into a rotary component, such as a shaft, as will be shown later.

[0063] FIG. 8 illustrates one of passive strut assemblies **618** which generally includes passive strut **626** and a torsional biasing spring **640**. Passive strut **626** includes a body segment **642** pivotably supported in passive strut pocket **606**, and a leg segment **644** having an engagement tip **646** configured to selectively engage first strut engagement surface **624** of ratchet teeth **622**. As is understood, passive strut **626** operates to normally establish a “coupled” state between inner race **602** and outer race **604** in a first rotary direction (arrow **650**) and a ratcheting or “freewheeling” state in a second rotary direction (arrow **652**). Referring back to FIG. 7, the passive one-way clutch is configured to provide a “dual” passive strut

engagement arrangement. Specifically, two pairs of diametrically opposed passive strut assemblies **618** are provided, with one pair shown engaged and the other pair shown ready to engage. This arrangement reduces the required backlash angle of ratchet teeth **622**. Passive struts **626** are meant to always be engaged with ratchet teeth **622**. If the speed differential between inner race **602** and outer race **604** works in the engagement direction (arrow **650**), then the passive one-way clutch is engaged and transfers torque. In contrast, if the speed of outer race **604** exceeds that of inner race **602** (arrow **652**), the passive one-way clutch is released and passive struts **626** will ratchet over teeth **622**.

[0064] Referring again to FIG. 8, passive struts **626** are configured to be “tail-heavy”, which means that due to the rotational speed, leg segment **644** of passive struts **626** moves outward in a pivoting motion relative to passive strut pocket **606** toward its deployed/locked position shown. This tail-heavy configuration is provided by locating a spring pin **656** on body segment **642** in an offset orientation relative to the pivot axis of passive strut **626**. Torsion spring **640** has a looped portion **660** surrounding spring pin **656**, a first leg portion **662** engaging outer race **604**, and a second leg portion **664** engaging body segment **642** of passive strut **626**. Leg portions **662** and **664** of spring **640** act along the same plane to avoid “tipping” of passive strut **626**. This “non-centric” configuration results in passive struts **626** being naturally biased toward its deployed/locked position.

[0065] FIG. 7 shows a pair of diametrically-opposed active strut assemblies **620** pivotally mounted in active strut pockets **608** of outer race **604** and aligned in an opposite actuation orientation with respect to passive strut assemblies **618**. The number and positioning of active strut assemblies **620** is merely a design choice and not intended to limit the disclosure. FIG. 9 is an enlarged partial view of clutch assembly **600** to better illustrate the structure and

operation of active strut assemblies **620** which are shown to generally include active strut **630** and a torsional biasing spring **672**. Active strut **630** is a “tip-heavy” configuration having a body segment **674** supported in active strut pocket **608**, and an actuation leg segment **676** having an end tip portion **678** configured to lockingly engage second strut engagement surface **628** of ratchet teeth **622**. Torsion spring **672** includes a loop section **680** surrounding a “centric” spring post **682** formed on active strut **630**, a first leg section **684** engaging outer race **604**, and a second leg section **686** having a bent end retained in a spring bore formed in leg segment **676** of active strut **630**. As outer race **604** rotates, tip portion **678** is centrifugally biased in a direction away from engagement with ratchet teeth **622** so as to naturally bias active strut **630** toward its non-deployed position (shown). Spring **672** is provided to assist in biasing strut **630** toward its non-deployed position and further assists in releasing active strut **630** from engagement with second strut engagement surface **628** of ratchet teeth **622**.

[0066] Active struts **630** work in cooperation with always-engaged passive struts **626**. When deployed (via energization of coil unit **614**), active struts **630** function to lock inner race **602** to outer race **604** in both directions, ensuring the second, active mode for bi-directional clutch assembly **600**, that is the Lock-Lock mode. The primary reason that only one pair of active struts **630** is used compared to two pair of passive struts **626** is that passive struts **626** engage first and then active struts **630** subsequently engage to provide the Lock-Lock mode. Active struts **630** never engage first, and consequently, they never have to meet strict backlash requirements for tooth engagement. Obviously, further combinations of this orientation/relationship can be used for higher torque capacity applications.

[0067] Referring now to FIGS. 10-12, armature ring **616** is shown in a non-actuated position relative to coil unit **614** and active struts **630**. Armature ring **616** is normally

biased toward its non-actuated position via a return spring 700 acting between an outer face surface 702 of outer race 604 and an inner face surface 704 of armature ring 616. Armature ring 616 is a magnetic component having a ring segment 706 overlying coil unit 614, and a lip flange segment 708 from which a pair of strut actuation features, shown as lugs 710 extend. Each strut actuation lug 710 on armature ring 616 is aligned with a corresponding leg segment 676 on active struts 630. As noted previously, armature ring 616 is fixed for common rotation with outer race 604 but is axially moveable relative thereto between its non-actuated position and an actuated position. FIG. 11 schematically illustrates that energization of coil unit 614 functions to generate a magnetic flux circuit or path 720 that is operable for magnetically attracting armature ring 616 toward coil unit 614 for causing movement thereof from its non-actuated position toward its actuated position, as is indicated by arrow 772.

[0068] Referring now primarily to FIGS. 12-14, selective actuation of the controllable one-way clutch of bi-directional clutch 600 will be detailed. Specifically, when coil unit 614 is energized, it causes armature ring 616 to initiate axial movement from its non-actuated position (FIG. 12) toward its actuated position, in opposition to the biasing of return spring 700. Strut actuation lugs 710 on armature ring 616 include a ramped profiled surface 740 selected based on the relationship between the range of axial movement of armature ring 616 and the corresponding arcuate range of pivotal motion of active struts 630 resulting from ramped profiled surface 740 engaging a top surface 742 of actuation leg segment 676 of active strut 630. Ramped profile surface 740 is preferably non-linear and may, without limitation, have a helical or arcuate configuration capable of pivotably driving active strut 630 from its non-deployed position to its fully-deployed position as a result of axial movement of armature ring 616 from its non-actuated position to its actuated position. FIG. 13 illustrates engagement

of ramped profile surface **740** on strut actuation lug **710** with top surface **742** on active strut **630** causing pivotal movement of active strut **630** in opposition to the biasing of torsion spring **672**. As armature ring **616** continues to get closer to coil unit **614**, it will continue to pivot active strut **630** toward its fully-deployed position whereat tip segment **678** is locked against surface **628** of ratchet tooth **622**. When power is turned off and coil unit **614** is de-energized, torsion spring **700** will forcibly move armature ring **616** axially back to its non-actuated position which in turn allows strut spring **672** to drive active strut **630** back to its non-deployed position, thereby releasing clutch assembly **600** from its Lock-Lock mode.

[0069] While bi-directional clutch assembly **600** is shown to include a passive one-way clutch and a controllable one-way clutch to provide Freewheel/Lock and Lock-Lock modes of operation, an alternative arrangement could be configured to replace the passive strut assemblies **618** with a pair of second active strut assemblies, in addition to the original pair of first active strut assemblies **620**. In such a variant, a second electromagnetic actuator (coil, etc.), operated independently from the first coil unit **614**, would be used to selectively actuate the second active struts. In such an alternative variant, the available modes would now include Freewheel, Lock-Lock, Ratchet Clockwise, and Ratchet Counterclockwise. Furthermore, clutch assembly **600** could be configured without the passive one-way clutch such that the controllable one-way clutch functions to shift between a Freewheel-Freewheel state (active strut **630** in non-deployed position) and a Lock-Ratchet state (active strut **630** in deployed position).

[0070] FIG. 15 is a schematic illustration of a disconnect arrangement **790** with bi-directional clutch assembly **600** operably disposed between a rotary input **800** and a rotary output **802**. In this arrangement, inner race **602** is fixed for common rotation with rotary output

802 while outer race **604** is fixed for common rotation with rotary input **800**. Bearings **804** support rotary output **802** for rotation relative to rotary input **800** about a common rotary axis. An ECU **810** functions to energize coil unit **614** and cause movement of armature ring **616** when it is desired to establish the Lock-Lock mode. This arrangement is well-suited for use as a disconnect clutch in an axle assembly of the type used in 4WD/AWD vehicles.

[0071] FIG. 16 is a schematic illustration of an electric drive axle (EDA) assembly **792** for a hybrid/electric vehicle and which generally includes an electric motor **820**, a geartrain **822**, a differential unit **824**, and a pair of axleshafts **826**, **828**, in addition to bi-directional clutch assembly **600**. In this configuration, a motor shaft **830** drives inner race **602** of clutch assembly **600** while outer race **604** drives an input gearset **832** of geartrain **822**. An output gearset **834** is driven by input gearset **832** and, in turn, drives differential unit **824** for transferring drive torque to the wheels via axleshafts **826**, **828**. Geartrain **822** can be a single-speed reduction unit (shown) or a multi-speed variant with clutch assembly **600** disposed in any such variant between motor shaft **830** and input gearset **832** to provide the connect/disconnect function therebetween.

[0072] FIG. 17 is an alternate embodiment of an electric drive axle assembly **792A**. In this arrangement, motor **820** drives first gearset **832** while bi-directional clutch **600** is located between first gearset **832** and second gearset **834** of geartrain. In this arrangement, outer race **604** acts as the input member while inner race **602** acts as the output member driving second gearset **834** which, in turn, drives differential unit **824**.

[0073] Referring to FIGS. 18-32, a controllable one-way clutch assembly **900**, constructed in accordance with the aspects of the present disclosure, will now be described in detail. While not limited to any particular vehicular application, clutch assembly **900** is well-

suited for use as a disconnect clutch in drivelines, and particularly in E-drive assemblies of the type previously mentioned. FIG. 18 is a sectional isometric view of clutch assembly **900** illustrating its basic components and configuration to include a clutch module **902** and a power-operated actuator module **904**. Clutch module **902** generally includes an outer race assembly **906** rotatably driven by a rotary input component (input shell **908**), a first output shaft **910** fixed for common rotation with outer race assembly **906**, a rotary inner race **912** driving a second output shaft **914**, and a plurality of strut assemblies **915** supported for movement on outer race assembly **906** relative to ratchet teeth **917** formed on inner race **912**. Outer race assembly **906** is assembled from a non-limiting arrangement of components including a first OR member **916** and a second OR member **918** rigidly interconnected via a series of fastener **920**. First OR member **916** is a powered metal component forming strut pockets **922** (FIG. 28) within which strut assemblies **915** are supported while second OR member **918** is a stamped metal component. Radially-extending drive lug **924** formed on first OR member **916** are retained in drive slots **926** formed in input shell **908** such that outer race assembly **906** is coupled for common rotation with input shell **908**. A snap ring **928** is used to maintain the axial orientation between outer race assembly **906** and input shell **908**. A bearing assembly **927** and a journal sleeve **929** are shown supporting second output shaft **914** for rotation relative to first output shaft **910**.

[0074] One of strut assemblies **215** is best shown in FIG. 28 to include a strut **930** and a strut spring **932**. Strut **930** is shown to include a body segment **934** disposed in strut pocket **922** and an engagement segment **936** having a tip portion **938** configured to engage ratchet teeth **917** formed on inner race **912**. A pivot post **940** extending through a spring post **942** of body segment **934** functions to support strut **930** for pivotal movement between a

deployed position and a non-deployed position. Strut spring **932** functions to normally bias strut **930** toward its deployed position and includes a coiled segment **944** surrounding spring post **942**, a first tang segment **946** engaging first OR member **916**, and a second tang segment **948** disposed in a spring retention bore **950** formed in engagement segment **936** of strut **930**. Struts **930** are tail-heavy which means that, due to the rotational speed of outer race assembly **906**, body segment **934** will move into strut pocket **922** so as to drive engagement segment **936** toward its deployed position. Struts **930** are intended to normally operate in multiple engagements (i.e., at least two) in their deployed positions. Thus, if the relative speed differential between outer race assembly **906** and inner race **912** is in the engagement direction, clutch assembly **900** will engage and transfer torque from input shell **908** to second output shaft **914**. If, however, the speed of outer race assembly **906** is less than that of inner race **912**, tip portion **938** of struts **930** will ratchet over ratchet teeth **917**.

[0075] As will be detailed, power-operated actuator module **904** is normally operated in a first or “power-off” condition to mechanically engage and hold struts **930** in their non-deployed positions, thereby allowing free-wheeling between out race assembly **906** and inner race **912** in both directions of relative rotation to establish the Freewheel mode. However, shifting power-operated actuator module **904** into a second or “power-on” condition acts to release struts **930** so as to allow strut springs **932** to drive struts **930** to their deployed position and into engagement with ratchet teeth **917** to establish the Lock/Ratchet mode.

[0076] Power-operated actuator module **914** is best shown in FIGS. 19 and 20 to generally include a stationary electromagnetic actuator **960**, a magnetic armature **962**, a bearing assembly **964** disposed between electromagnetic actuator **960** and armature **962**, and a ballramp mechanism **966** disposed between armature **962** and an annular hub segment **968** of second OR

member **918**. Stationary electromagnetic actuator **960** includes an annular coil unit **970** encapsulated in a non-magnetic coil housing **972**. Housing **972** defines a cylindrical relief defining an armature cavity **974** configured to accommodate axial movement of armature **962**. Armature **962** is configured to include a cylindrical tube segment **980** generally aligned within armature cavity **974** in coil housing **972**, a radial flange segment **982** extending from tube segment **980**, an axially-extending rim segment **984** extending from flange segment **982**, and a plurality of blocking lug segments **986** extending radially from rim segment **984**. Bearing assembly **964** is shown located between coil housing **972** and tube segment **980** of armature **962**.

[0077] Ballramp mechanism **966** includes a cam ring **990** fixedly secured (i.e., pressed on, splined, etc.) to hub segment **968** of second OR member **918** for common rotation with outer race assembly **906**. Note that splines **992** formed on hub segment **968** mesh with splines (See FIG. 18) formed on a raised boss segment **994** of first output shaft **910** such that outer race assembly **906** drives first output shaft **910** in response to rotation of input shell **908**. Cam ring **990** defines a series of circumferentially-spaced helically-oriented cam grooves **1000**. Ballramp mechanism **966** further includes rollers, such as balls **1002**, disposed within cam grooves **1000**. Ballramp mechanism **966** also includes a corresponding series of circumferentially-spaced ramp grooves **1004** formed in rim segment **984** of armature **962**. Ramp grooves **1004** are aligned with cam grooves **1000** and retain balls **1002** therebetween.

[0078] The integration of ballramp mechanism **996** between armature **962** and second OR member **918** functions to cause both axial and rotational movement of armature **962** relative to outer race assembly **906** in response to energization (i.e., power-on) and de-energization (i.e. power-off) of coil unit **970**. This combined axial and rotational movement

defines a helical travel path for armature **962** between a first or “engaged” position when coil unit **970** is in its power-off condition and a second or “released” position when coil unit **970** is in its power-on condition. This helical movement of armature **962** functions to cause blocking lug segments **986** to engage and hold struts **930** in their non-deployed positions when armature **962** is located in its engaged position. In contrast, movement of armature **962** to its released position upon energization of coil unit **970** causes blocking lug segments **986** to disengage struts **930** and allow strut springs **932** to drive struts **930** into their deployed positions.

[0079] With coil unit **970** in its power-off condition, an armature biasing engagement **1010** is provided to drive armature **962** into its engaged position. Armature biasing arrangement **1010** is best shown in FIGS. 23 and 24 to include a coil spring **1012** disposed within a spring retainer cavity **1014** formed in cam ring **990**. A pass-thru window **1016** is formed in spring retainer cavity **1014**. A spring tab **1018** extending from armature **962** extends through window **1016** and into spring retainer cavity **1014**. A first end segment **1012A** of coil spring **1012** engages a first end wall **1014A** of spring retainer cavity **1014** while a second end segment **1012B** of coil spring **1012** engages spring tab **1018** on armature **962**. Upon coil unit **970** being shifted into its power-on condition, the magnetic field generated functions to drive armature **962** from its engaged position into its released position which, in turn, functions to compress coil spring **1012** and generate an armature return force. As noted, with armature **962** held in its released position during energization of coil unit **970**, blocker lug segments **986** are disengaged from struts **930**, thereby allowing struts **930** to move back into their deployed positions. Subsequent de-energization of coil unit **970** functions to release the armature return force and allow coil spring **1012** to drive armature **962** back to its engaged

position with blocker lug segment **986** again engaging and holding struts **930** in their non-deployed position.

[0080] With armature **962** located in its engaged position, an upper ramped surface **1030** formed on blocker lug segments **986** engages an underside surface **1032** on engagement segment **936**. The helical motion of armature **962** causes blocker lug segments **986** to raise struts **930** from their deployed position to their non-deployed positions. FIGS. 19, 20, 24, and 26-30 illustrate this engagement between upper ramped surface **1030** on blocker lugs segments **986** and underside surface **1032** on struts **930** when coil unit **970** is not energized so as to define the Freewheel mode and provide a “non-powered” disconnect function between input shell **908** and second output shaft **914**. Note that blocker lugs segments **986** are only partially disposed below struts **930** so as to promote a simple release upon subsequent energization of coil unit **970**. FIG. 30 illustrates a non-limiting example of a preferred contour for ramped surface **1030**.

[0081] FIGS. 18, 31 and 32 illustrate movement of armature **962** into its released position, in opposition to the biasing of armature spring **1012**, in response to coil unit **970** being energized into its power-on condition. Due to the helical trajectory armature **962** follows, it is both axially retracted from, and rotated relative to, struts **930** in response to the magnetic field generated by coil unit **970**. The axial vector of this helical trajectory is associated with retraction of tube segment **980** of armature **962** relative to armature cavity **974**. Likewise, the rotational vector of this helical trajectory is generated by ballramp mechanism **966** in response to axial retraction of armature **962**. The characteristic (i.e. depth, width, angulation, length, etc.) of cam grooves **1000** and ramped grooves **1004** dictate the relationship between the axial and rotational components of the helical trajectory.

[0082] FIG. 26 illustrates a travel-limiting arrangement configured to properly locate armature **962** in each of its engaged and released positions. Specifically, a post **1050** extending axially from cam ring **990** is retained in an elongated arcuate slot **1052** formed in flange segment **982** of armature **962**. With armature **962** located in its engaged position (shown), post **1050** engages a first end of slot **1052**. In contrast, when armature **962** is located in its released position, post **1050** engages the opposite end of slot **1052**. The biasing force exerted by armature spring **1012** on armature **962**, when armature **962** is located in its engaged position is selected to work cooperatively with the linear-to-rotary conversion characteristic of ballramp mechanism **966** to assure that struts **930** are fully located in their non-deployed positions to avoid unintended contact with ratchet teeth **917**.

[0083] Clutch assembly **900** is distinct in its configuration from each of the previously disclosed clutch assemblies shown in FIGS. 1-14 in that power-operated actuator **904** is not employed to deploy an active strut. Rather, clutch assembly **900** utilizes power-operated actuator **904** to block deployment of passive struts **930** in an effort to provide the power-off disconnect feature and a “power-on” connect feature. However, those skilled in the art can recognize the ability to install clutch assembly **900** into any of the E-drive arrangements in FIGS. 16 and 17 and/or into the driveline disconnect arrangement in FIG. 15.

[0084] In general, clutch assembly **900** is comprised of an outer race including pockets for a plurality of ratchet-type passive struts, an electromagnetic actuator including a stationary coil unit and a moveable armature which rotates with the outer race, a ballramp mechanism, and an inner race having ratchet teeth normally engaged by the passive struts to provide a Lock/Ratchet mode of operation.

[0085] The outer race is an assembly comprised of a PM first outer race member formed to include all net-shaped features (i.e., strut pockets, spring connections, etc.) and a stamped second outer race member configured to transfer torque from the input component to an output component. Preferably, the input component is a rotary, large diameter shell fixedly coupled to the outer race assembly while the output component is a shaft coupled for rotation with the second outer race member. The passive struts are arranged to work in multiple engagements with at least two of the passive struts always engaged with the ratchet teeth on the inner race. A strut spring normally biases each of the passive struts toward its fully deployed position for maintaining low speed engagement. The passive struts are tail-heavy to drive the passive struts toward their fully deployed positions at higher rotary speeds of the outer race assembly.

[0086] The moveable armature has strut blocking features configured to engage and hold the passive struts in a tucked, non-deployed position when the armature is located in an engaged position while the coil unit is powered off. An armature biasing spring functions to bias the armature toward its engaged position. Upon the occurrence of a power-on condition, the armature moves to a released position due to its attraction to the coil unit's magnetic field. This movement of the armature between its engaged and released positions is not linear (i.e., axial or back-and-forth), but rather the armature moves in a helical trajectory due to integration of ballramp mechanism between the outer race assembly and the armature. Thus, movement of the armature from its engaged position to its released position causes the strut blocking features to retract axially away from the struts as well as rotate to a location behind the strut. This allows the struts to move back to their fully deployed positions. When the coil unit is subsequently powered off, the armature extends axially and rotates toward the struts, whereby

the strut blocking features engage the struts and mechanically move them from their fully deployed position into their non-deployed position. The coil and armature actuation arrangement can be driven by several magnetic field concepts. The helical armature trajectory provided by integration of the ballramp unit provides advantages in armature movement by combining axial and rotary motion.

[0087] The foregoing description of the embodiments has been provided for purposes of illustration and description. It is not intended to be exhaustive or to limit the disclosure. Individual elements or features of a particular embodiment are generally not limited to that particular embodiment, but, where applicable, are interchangeable and can be used in a selected embodiment, even if not specifically shown or described. The same may also be varies in many ways. Such variations are not to be regarded as a departure from the disclosure, and all such modifications are intended to be included within the scope of disclosure.

CLAIMS

What is claimed:

1. A controllable one-way clutch assembly operably disposed between a rotary input member and a rotary output member, the clutch assembly comprising:

a first clutch component coupled for rotation with the input member and defining a plurality of strut pockets;

a second clutch component coupled for rotation with the output member and defining ratchet teeth;

a plurality of struts each mounted in a corresponding one of the strut pockets for movement between a deployed position engaged with the ratchet teeth and a non-deployed position disengaged from the ratchet teeth;

a plurality of strut springs each associated with a corresponding one of the struts and configured to bias the strut toward its deployed position; and

a power-operated actuator operable in a power-off condition to mechanically engage and hold each of the struts in its non-deployed position and further operable in a power-on condition to disengage each of the struts and allow the strut springs to drive the struts to their deployed positions.

2. The clutch assembly of Claim 1, wherein the power-operated actuator includes a stationary coil unit, an armature moveable relative to the coil unit and the struts between an engaged position whereat the armature engages and holds the struts in their non-deployed positions and a release position whereat the armature is released from engagement with the struts, an armature biasing arrangement configured to bias the armature toward its engaged

position, and a ballramp mechanism operable to move the armature in a helical trajectory between its engaged and released positions.

3. The clutch assembly of Claim 2, wherein the armature includes strut blocking features arranged to engage and hold each of the struts in its non-deployed position when the armature is located in its engaged position and the coil unit is operating in its power-off condition, and wherein the helical trajectory of the armature causes the strut blocking features to disengage the struts when the armature is moved to its released position in response to the coil unit being shifted into its power-on condition.

4. The clutch assembly of Claim 3, wherein the strut blocking features are radially-extending lugs extending from the armature.

5. The clutch assembly of Claim 2, wherein the ballramp mechanism is disposed between the armature and the first clutch component.

6. The clutch assembly of Claim 5, wherein the ballramp mechanism includes a cam ring fixed to the first clutch member and having a cam groove formed therein, a ramp groove formed in the armature, and a ball disposed between the cam groove and the ramp groove, and wherein the cam groove and ramp groove are aligned such that axial movement of the armature relative to the coil unit causes corresponding rotary movement of the armature, thereby defining the helical trajectory of the armature.

7. The clutch assembly of Claim 6, wherein the armature defines an elongated slot, a post extends axially from the cam ring into the slot and is slideable within the slot, and wherein the rotary movement of the armature is limited by the post engaging first and second ends of the slot.

8. The clutch assembly of Claim 7, wherein when the armature is located in its engaged position the post engages the first end of the slot, and wherein when the armature is located in its released position the post engages the second end of the slot.

9. The clutch assembly of Claim 1, wherein the clutch assembly is a motor disconnect clutch installed in an electric transaxle or an electric drive axle to establish a power-off disconnect function between a motor driving the input member and a geartrain driving the output member.

10. A controllable one-way clutch assembly operably disposed between a rotary input member and a rotary output member, the clutch assembly comprising:

an outer race coupled for rotation with the input member and defining a plurality of strut pockets;

an inner race coupled for rotation with the output member and defining ratchet teeth;

a plurality of struts each mounted in a corresponding one of the strut pockets for movement between a deployed position engaged with the ratchet teeth and a non-deployed

position disengaged from the ratchet teeth, wherein each of the plurality of struts is biased toward its deployed position; and

a power-operated actuator moveable between a power-off condition engaging each of the struts in its non-deployed position and a power-on condition disengaging each of the struts and allowing the struts to move into their deployed positions.

11. The clutch assembly of Claim 10, wherein the power-operated actuator includes a stationary coil unit, an armature moveable relative to the coil unit and the struts between an engaged position whereat the armature engages and holds the struts in their non-deployed positions and a release position whereat the armature is released from engagement with the struts, an armature biasing arrangement biasing the armature toward its engaged position, and a ballramp mechanism operable to move the armature both axially and rotationally between its engaged and released positions.

12. The clutch assembly of Claim 11, wherein the armature includes strut blocking features arranged to engage and hold each of the struts in its non-deployed position when the armature is located in its engaged position and the coil unit is operating in its power-off condition, and wherein the rotational and axial movement of the armature causes the strut blocking features to disengage the struts when the armature is moved to its released position in response to the coil unit being shifted into its power-on condition.

13. The clutch assembly of Claim 12, wherein the strut blocking features are radially-extending lugs extending from the armature.

14. The clutch assembly of Claim 11, wherein the ballramp mechanism is disposed between the armature and the outer race.

15. The clutch assembly of Claim 14, wherein the ballramp mechanism includes a cam ring fixed to the first clutch member and having a cam groove formed therein, a ramp groove formed in the armature, and a ball disposed between the cam groove and the ramp groove, and wherein the cam groove and ramp groove are aligned such that axial movement of the armature relative to the coil unit causes corresponding rotational movement of the armature.

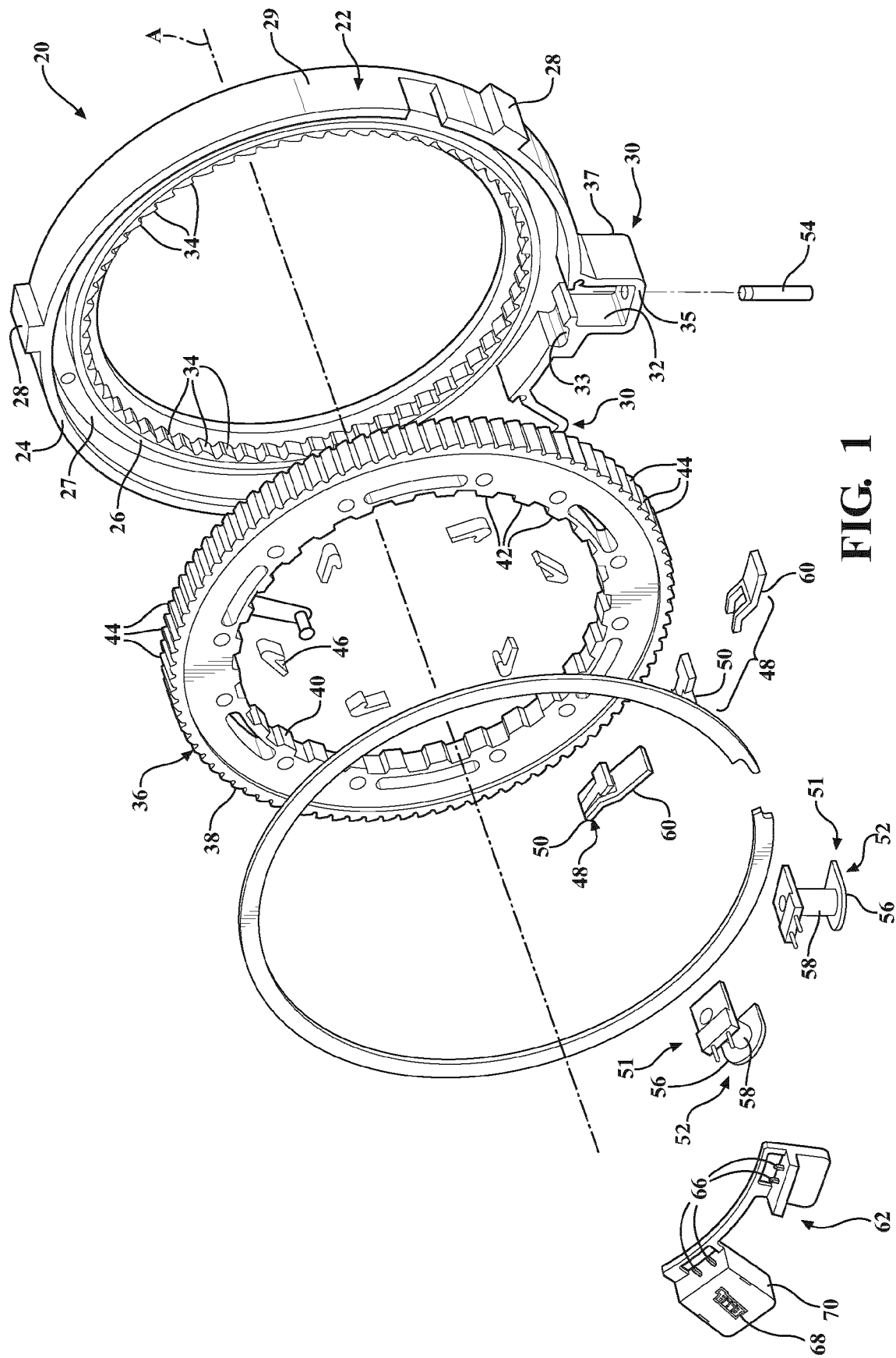


FIG. 1

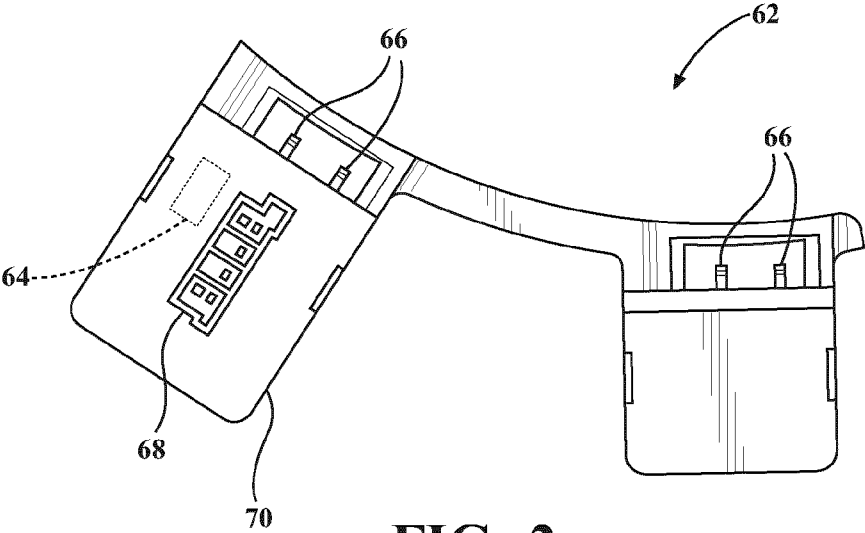
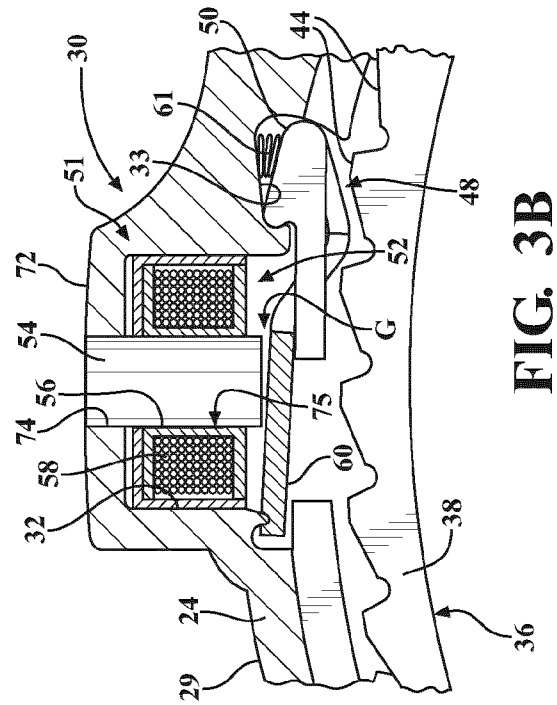
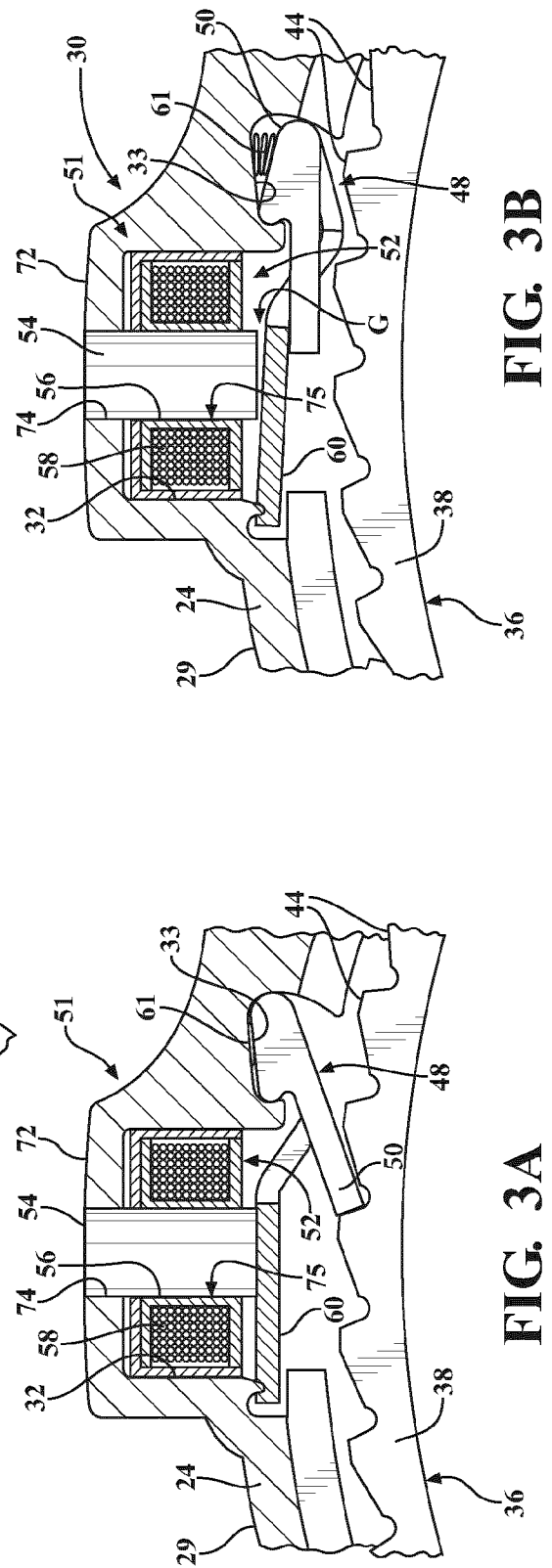
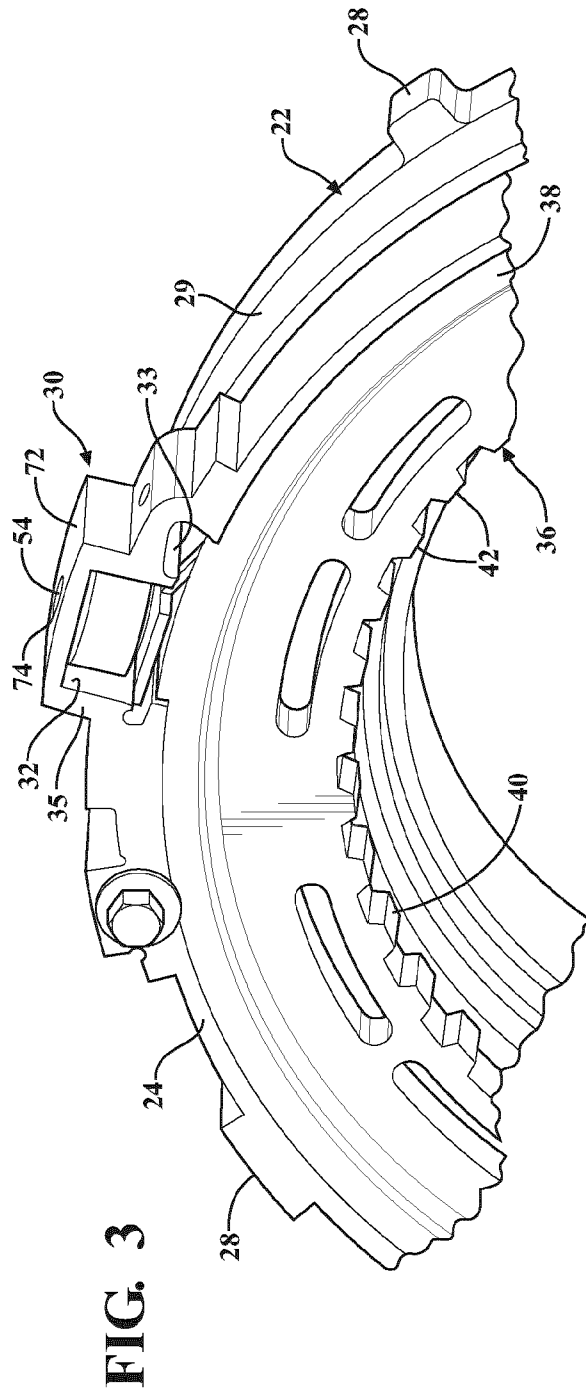


FIG. 2



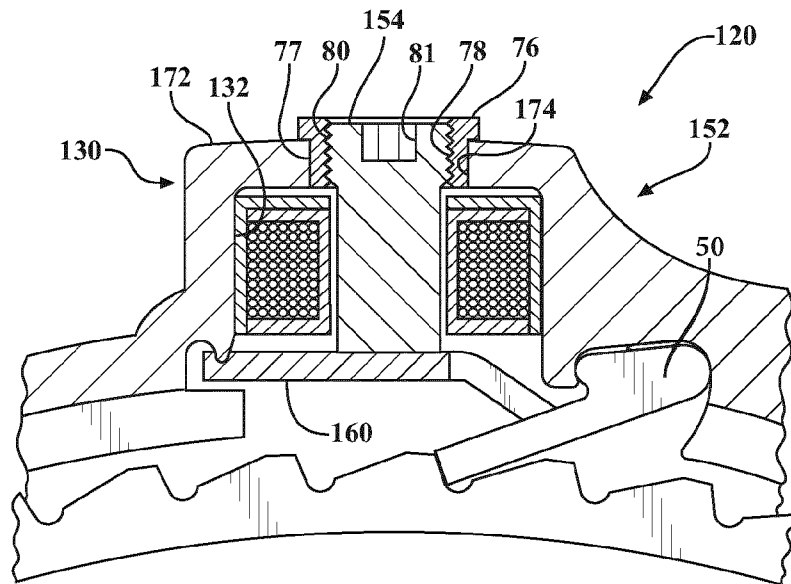


FIG. 4A

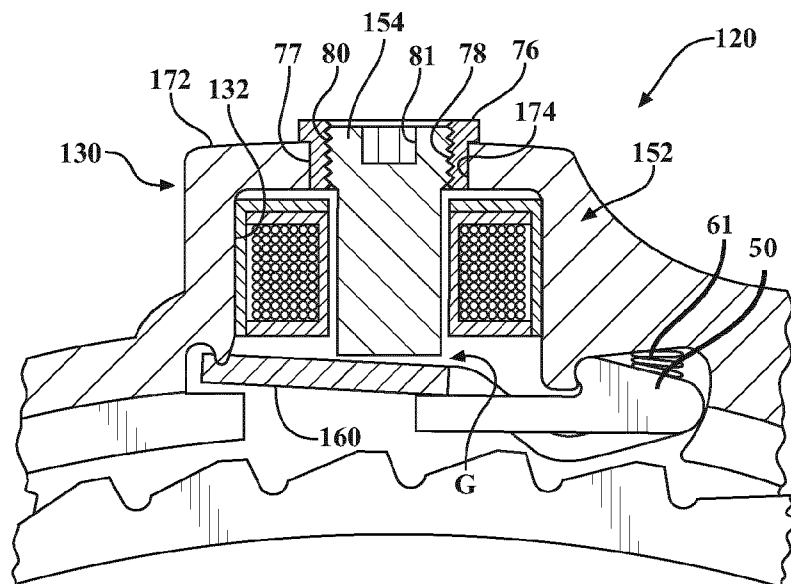


FIG. 4B

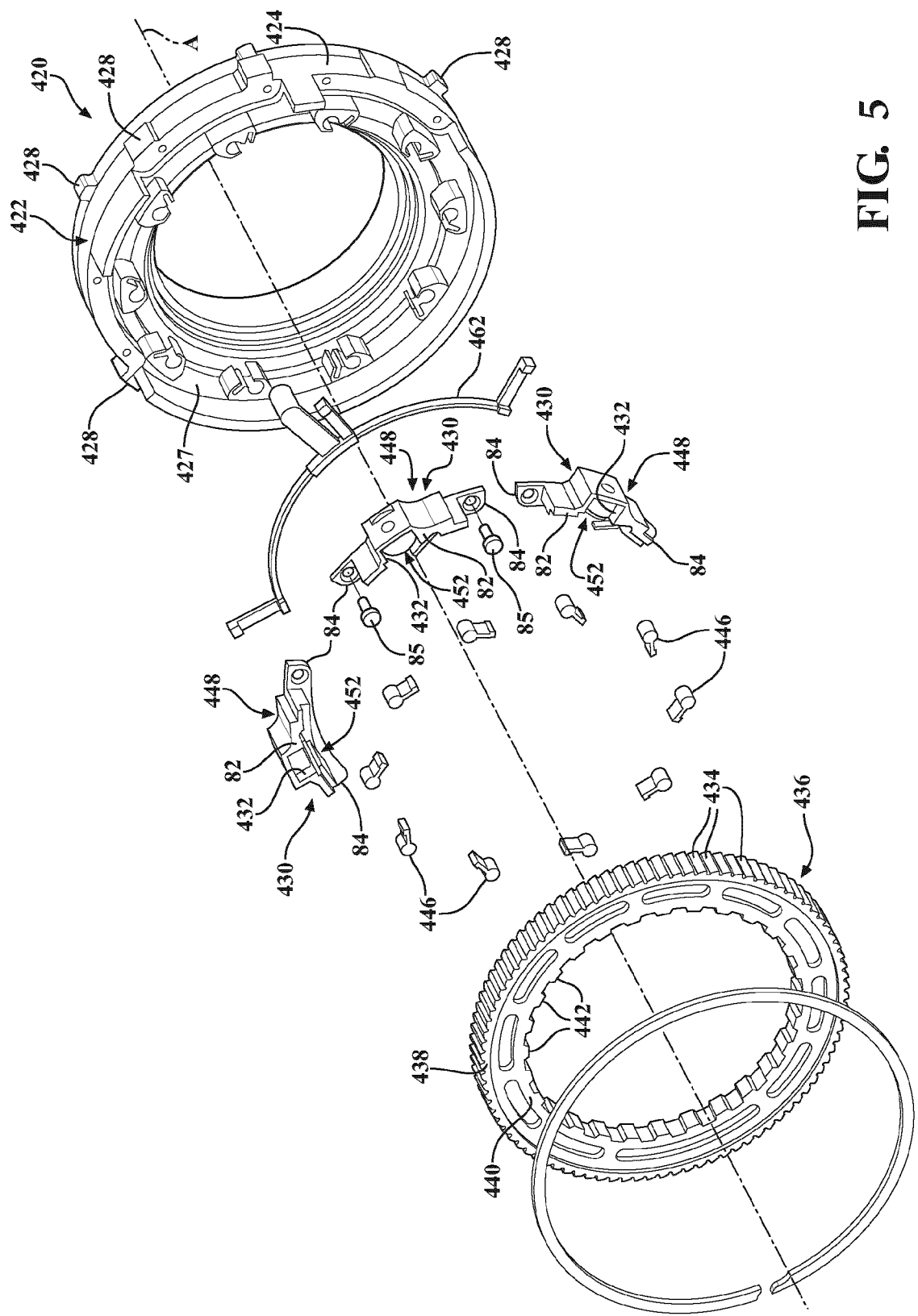
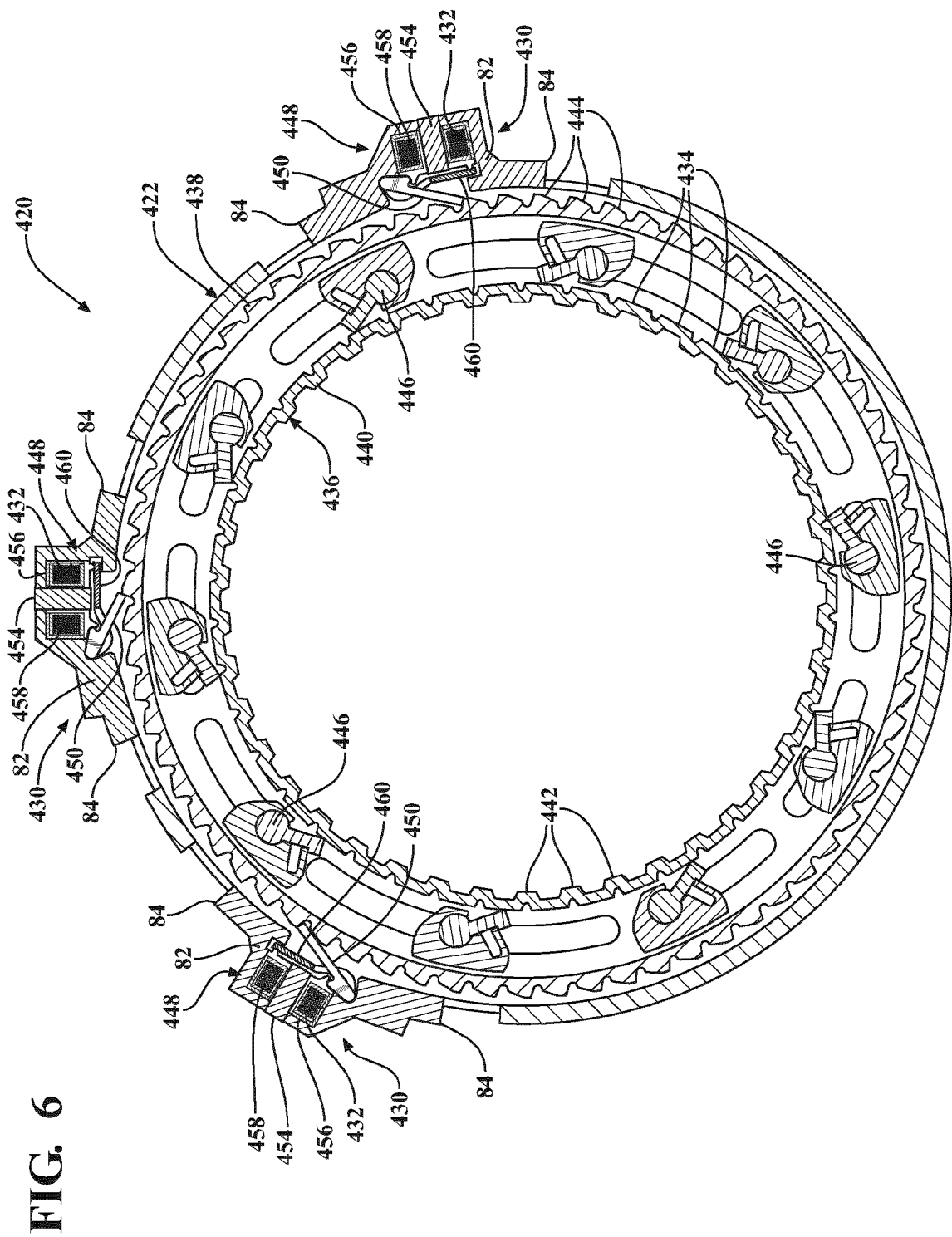


FIG. 5



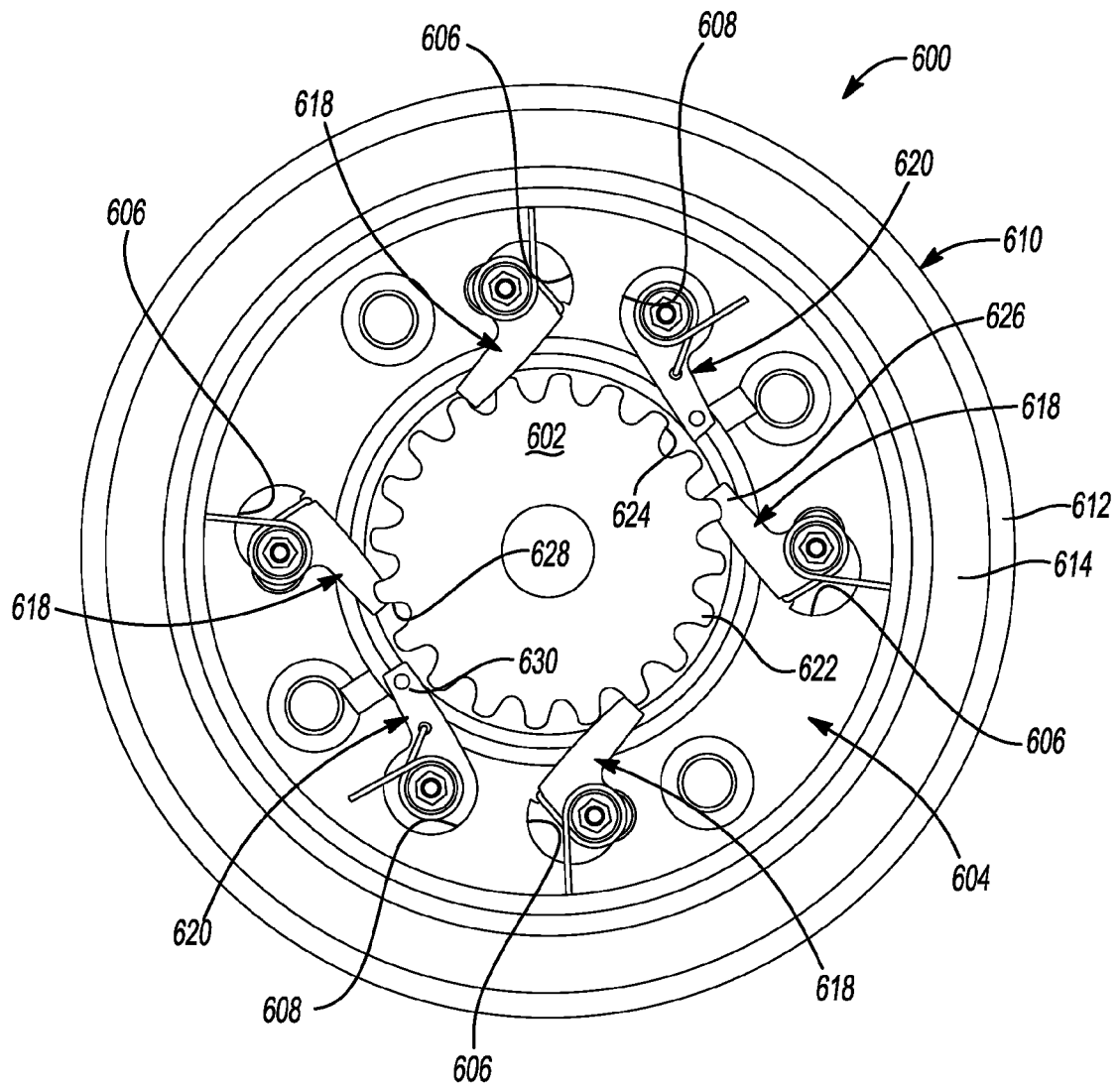


Fig-7

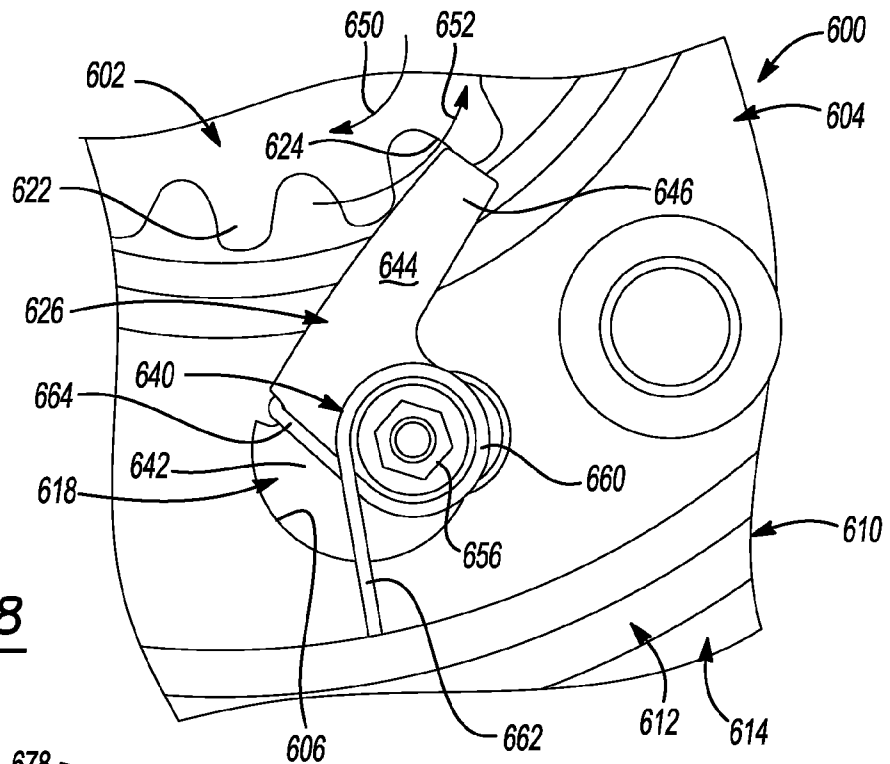


Fig-8

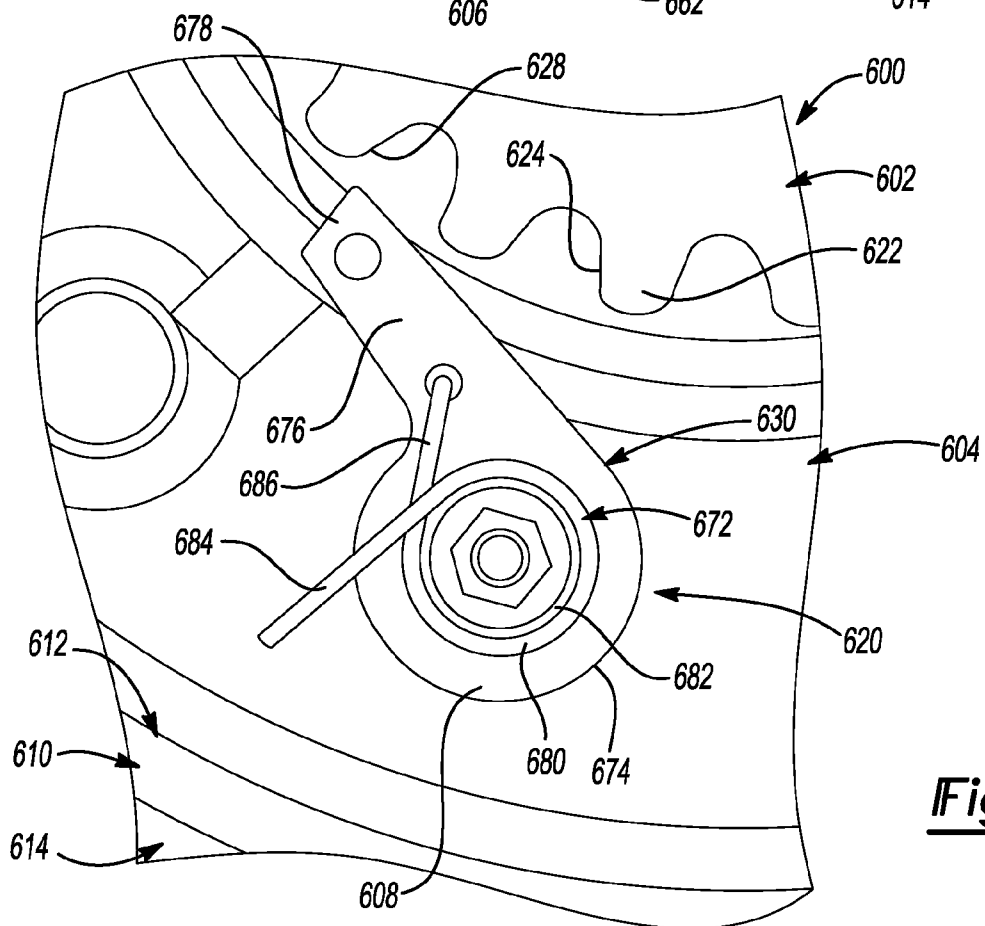


Fig-9

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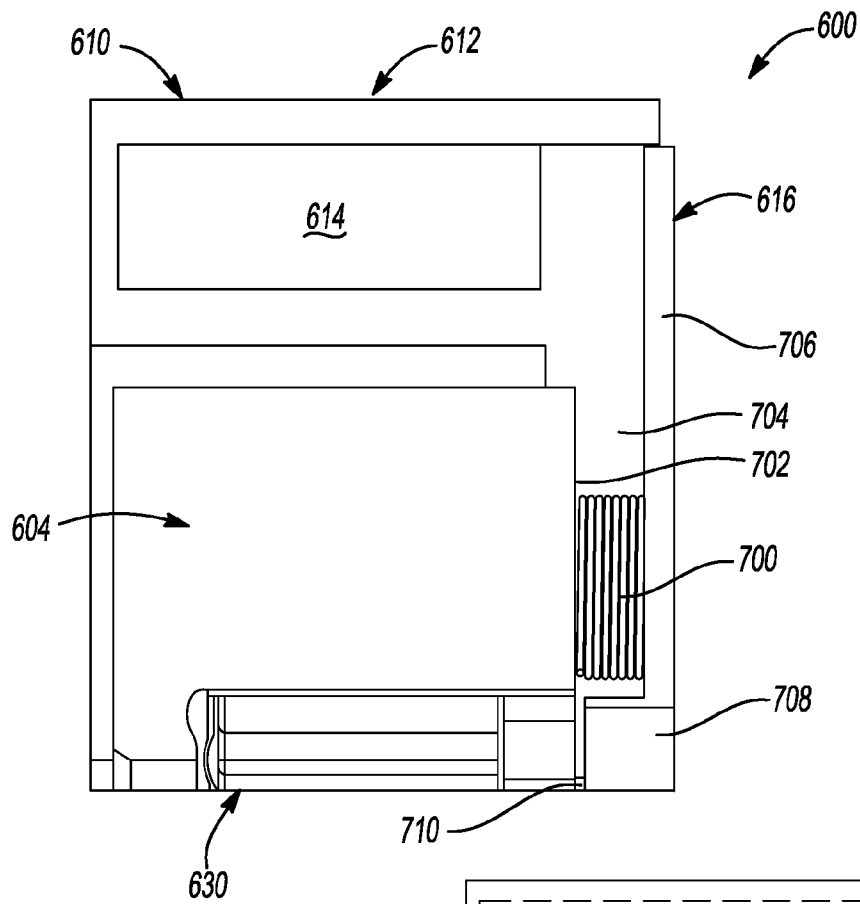
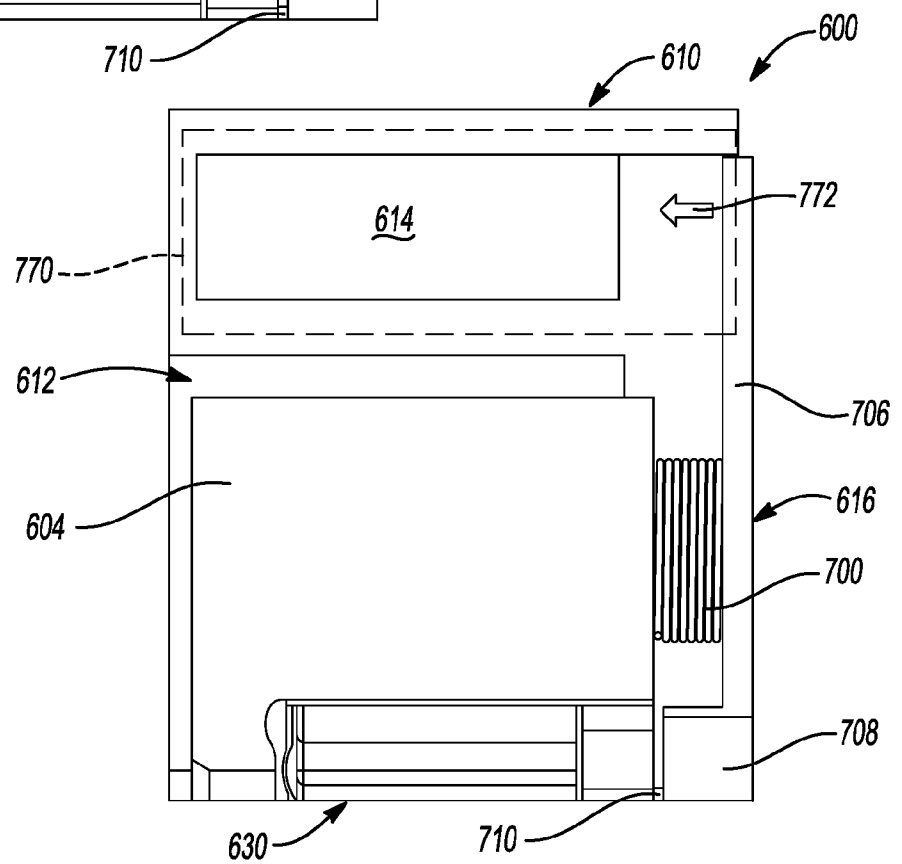


Fig-10

Fig-11



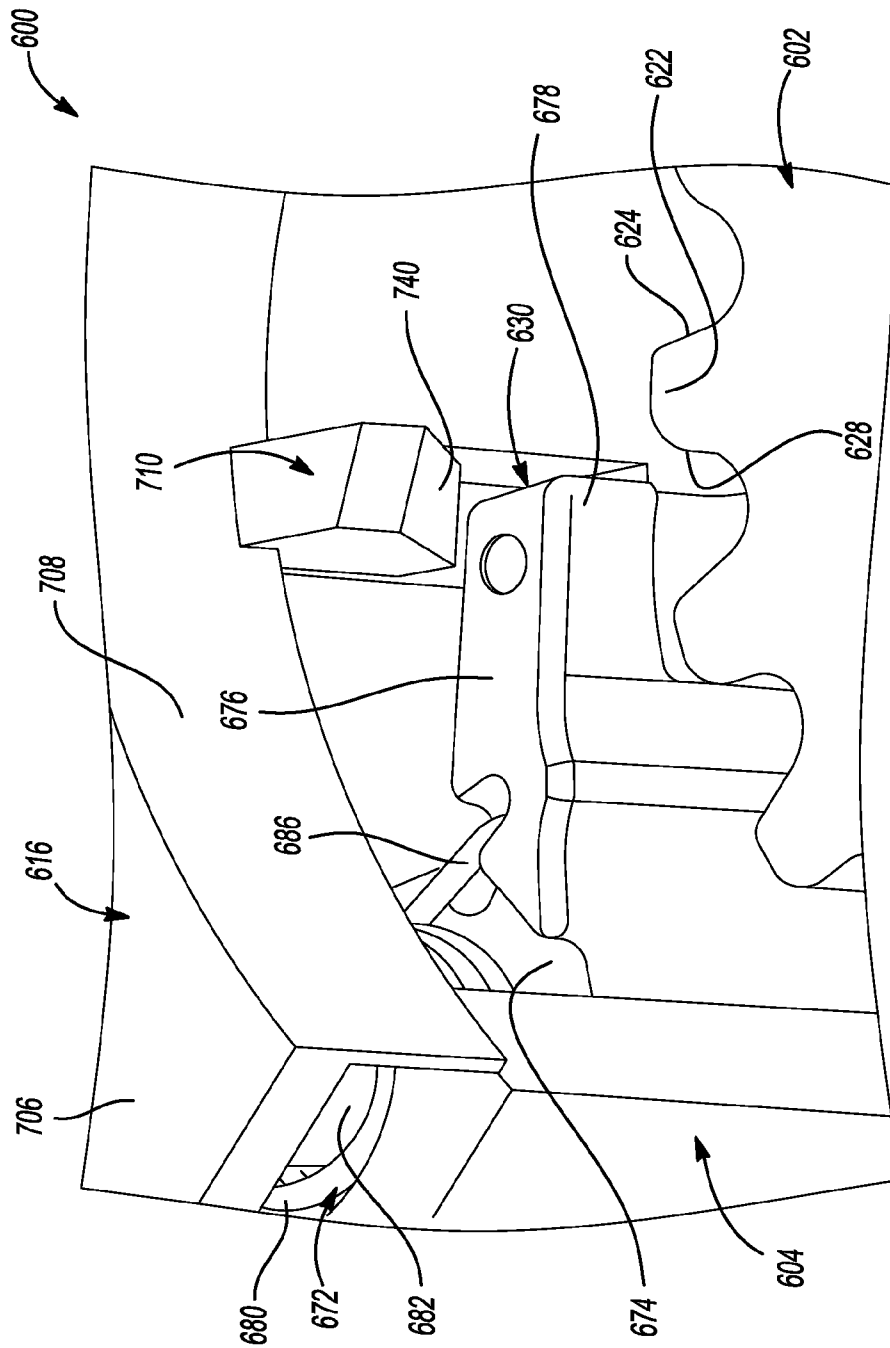


Fig-12

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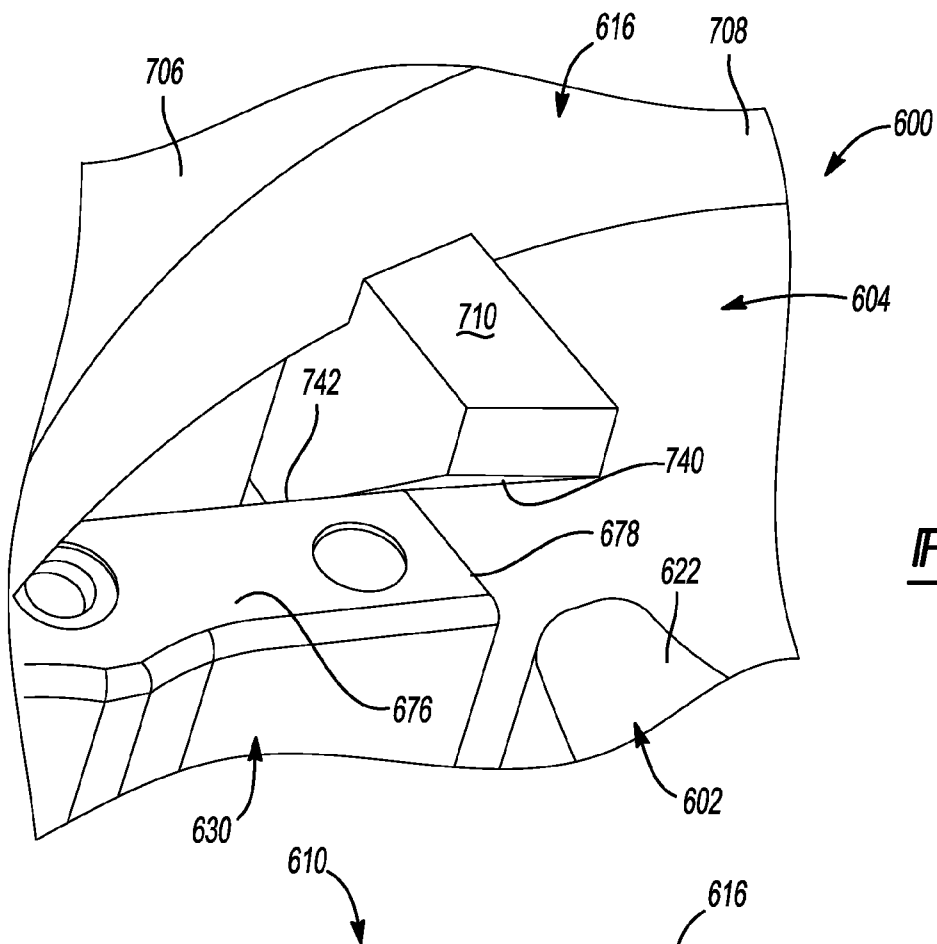


Fig-13

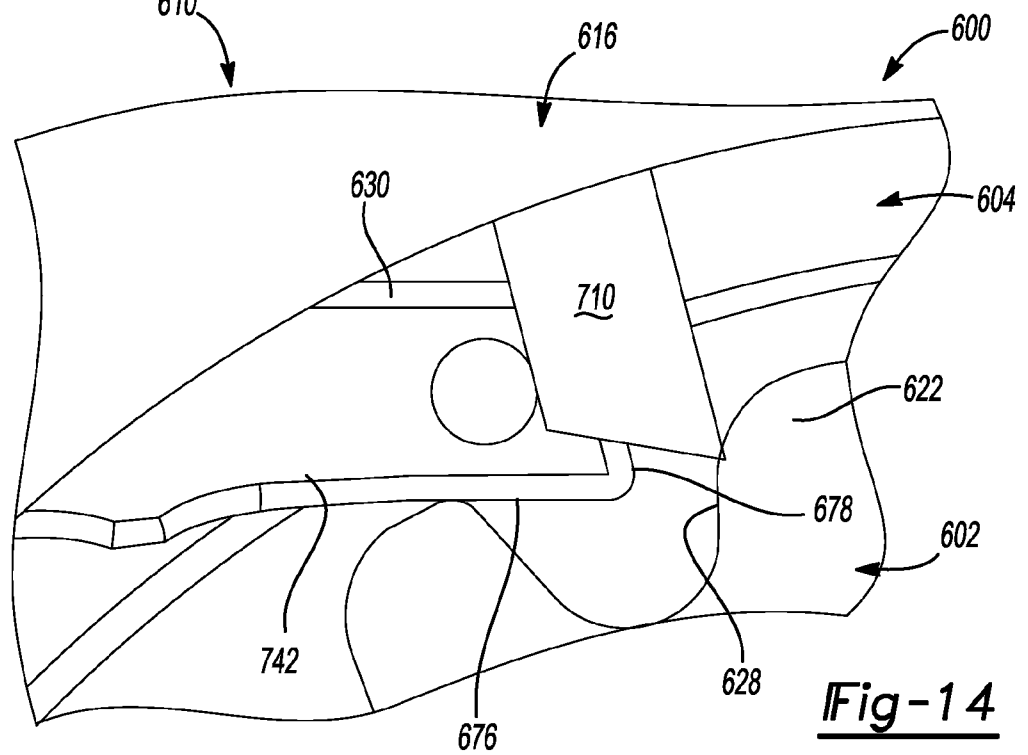
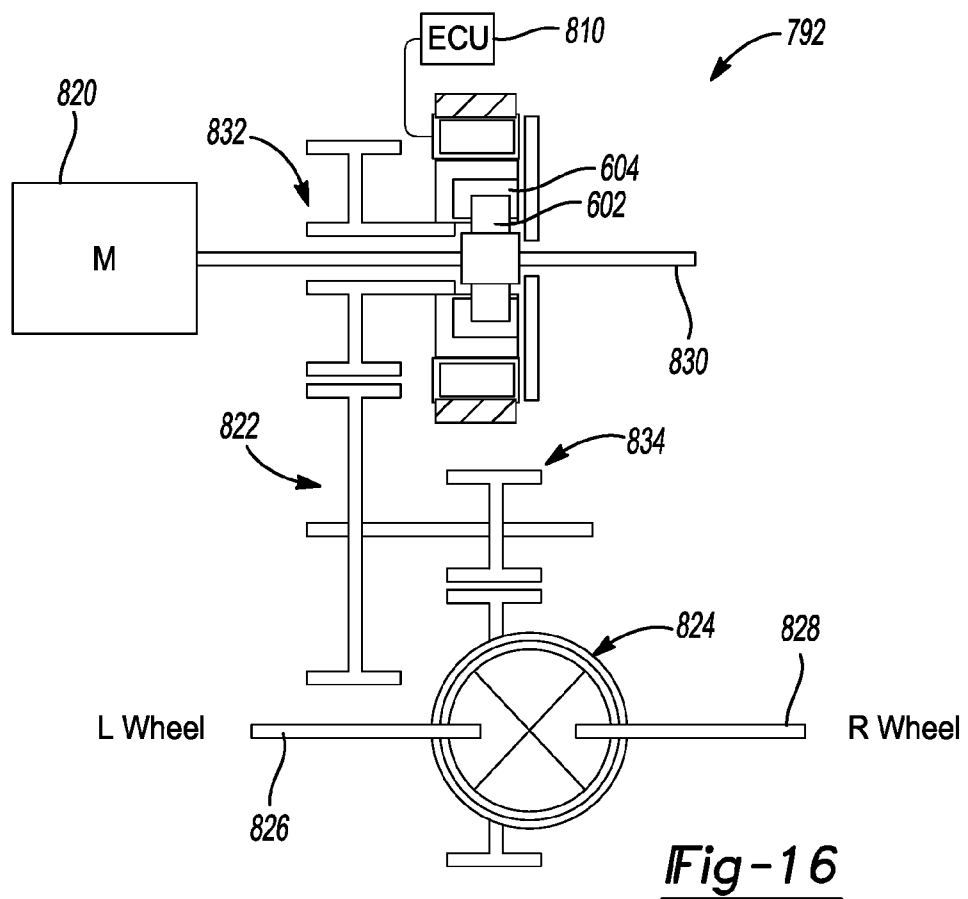
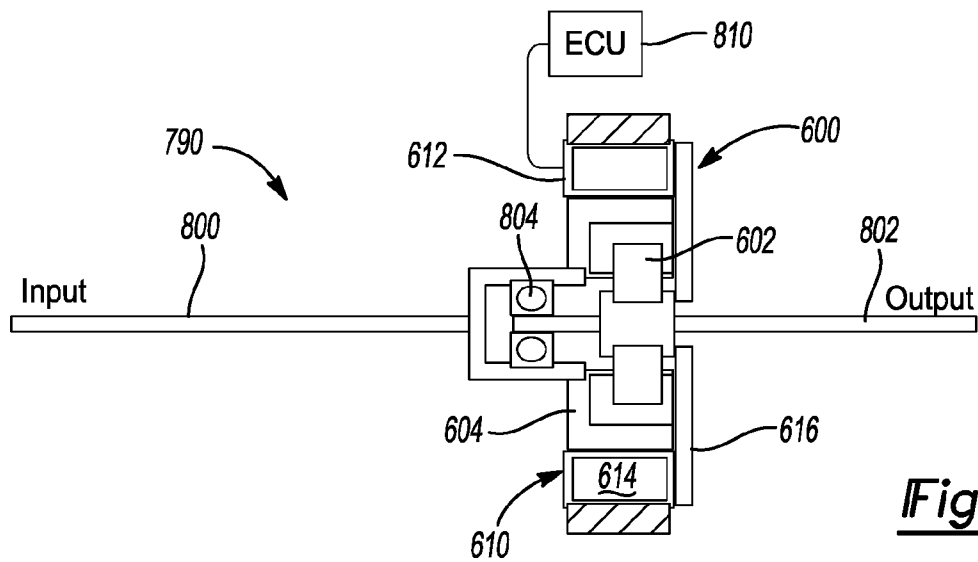


Fig-14

12/26



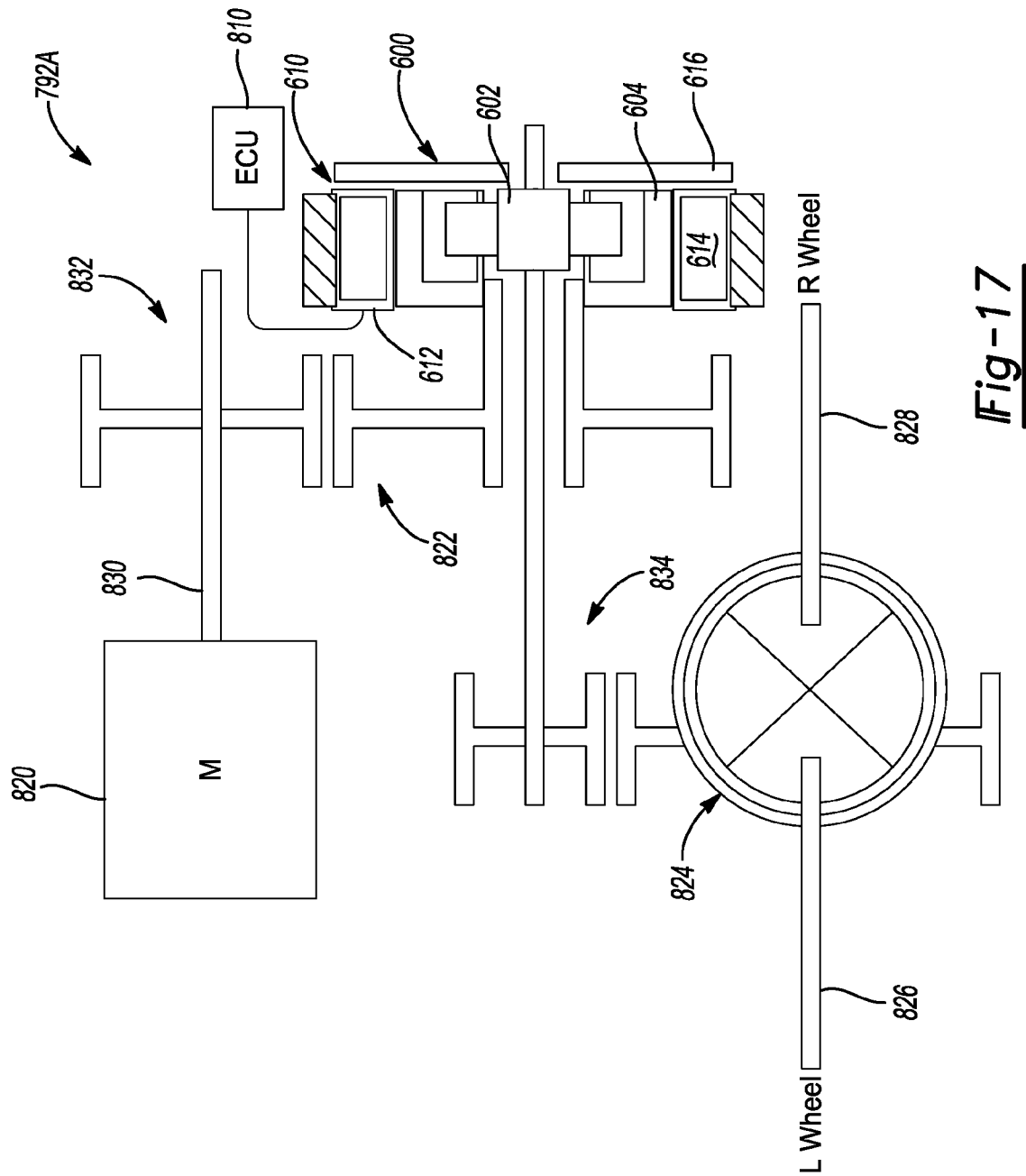
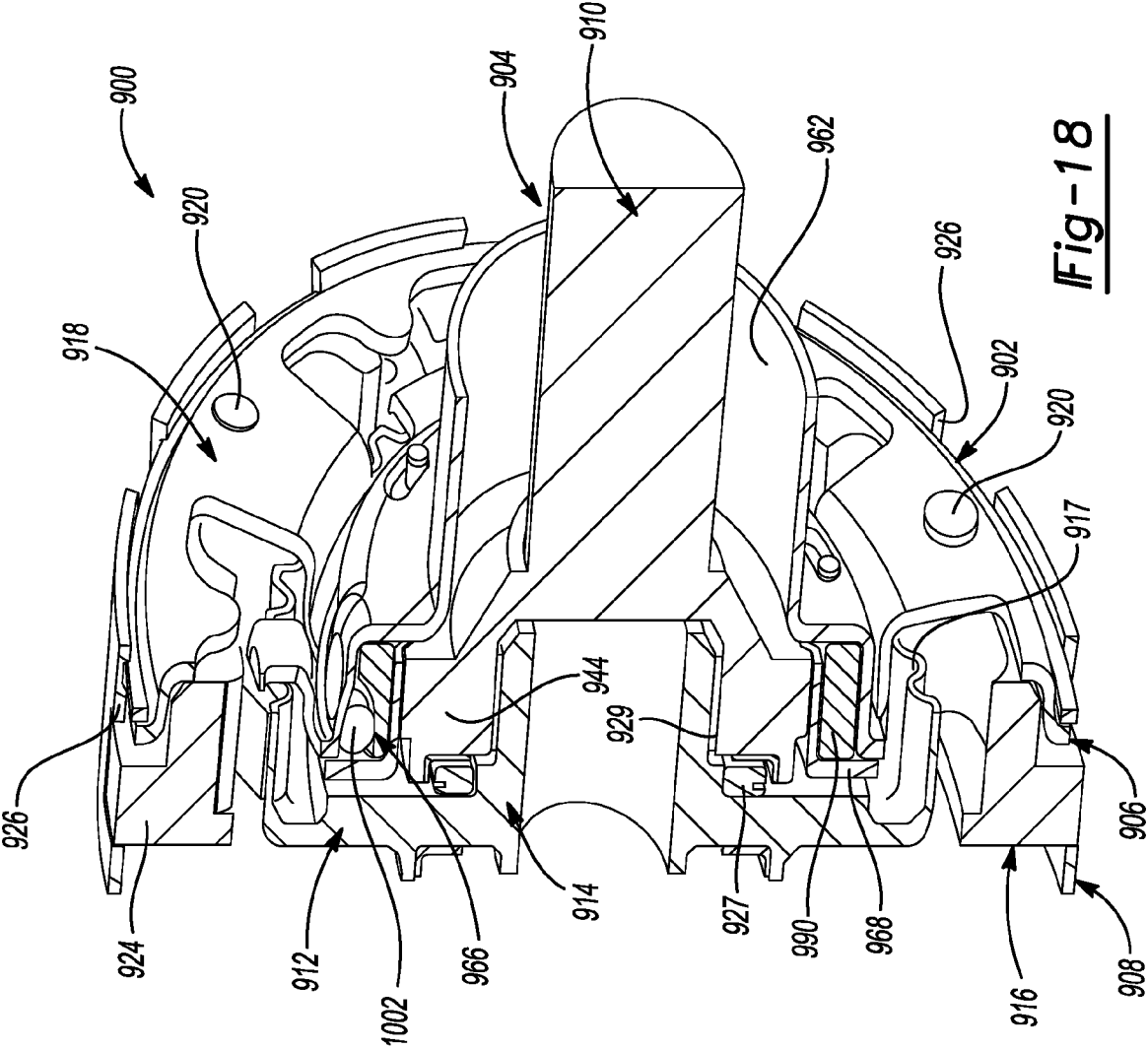
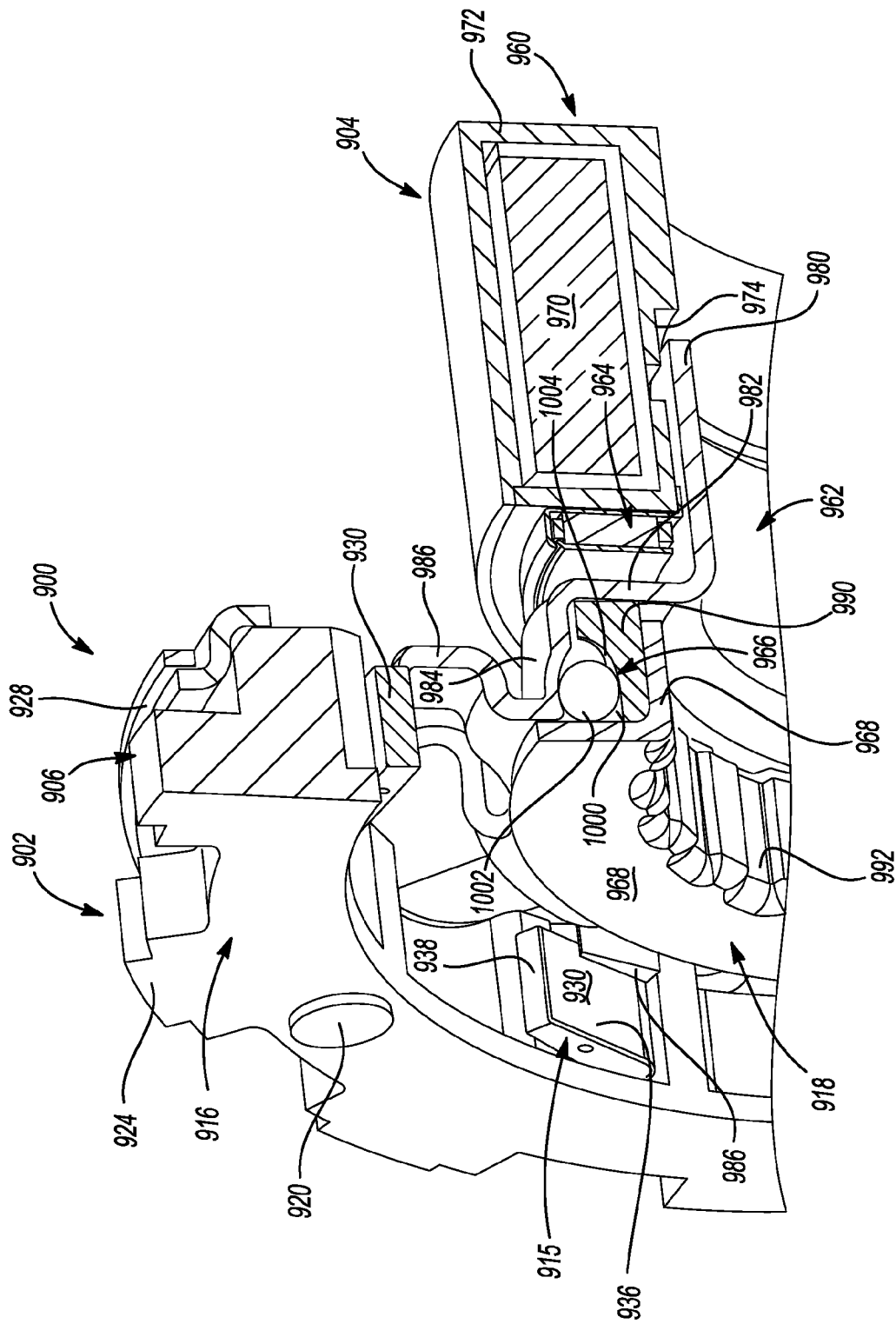


Fig-17





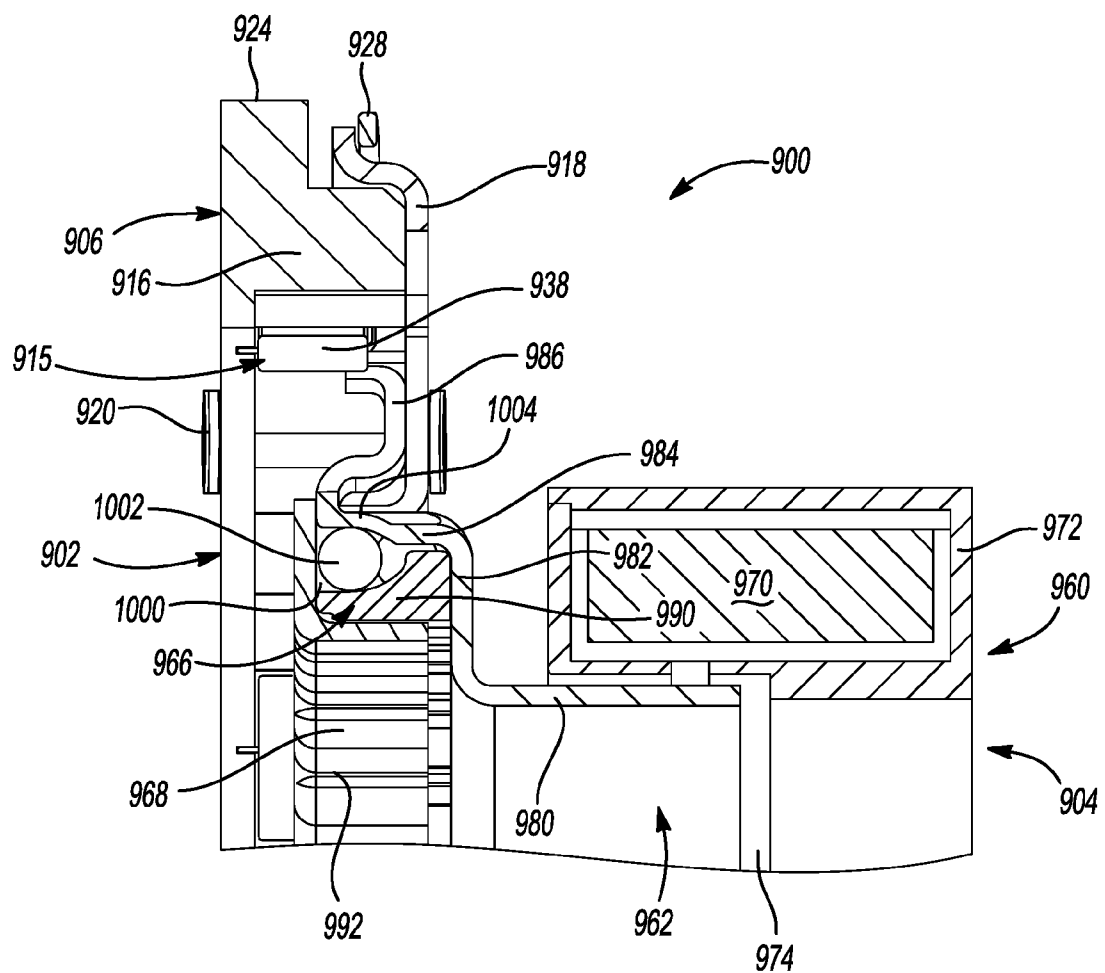
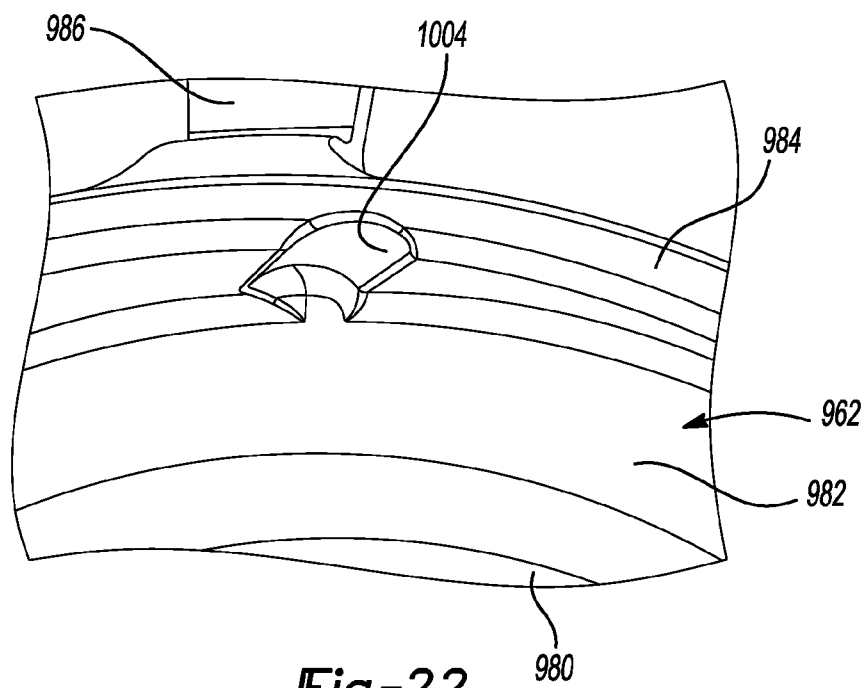
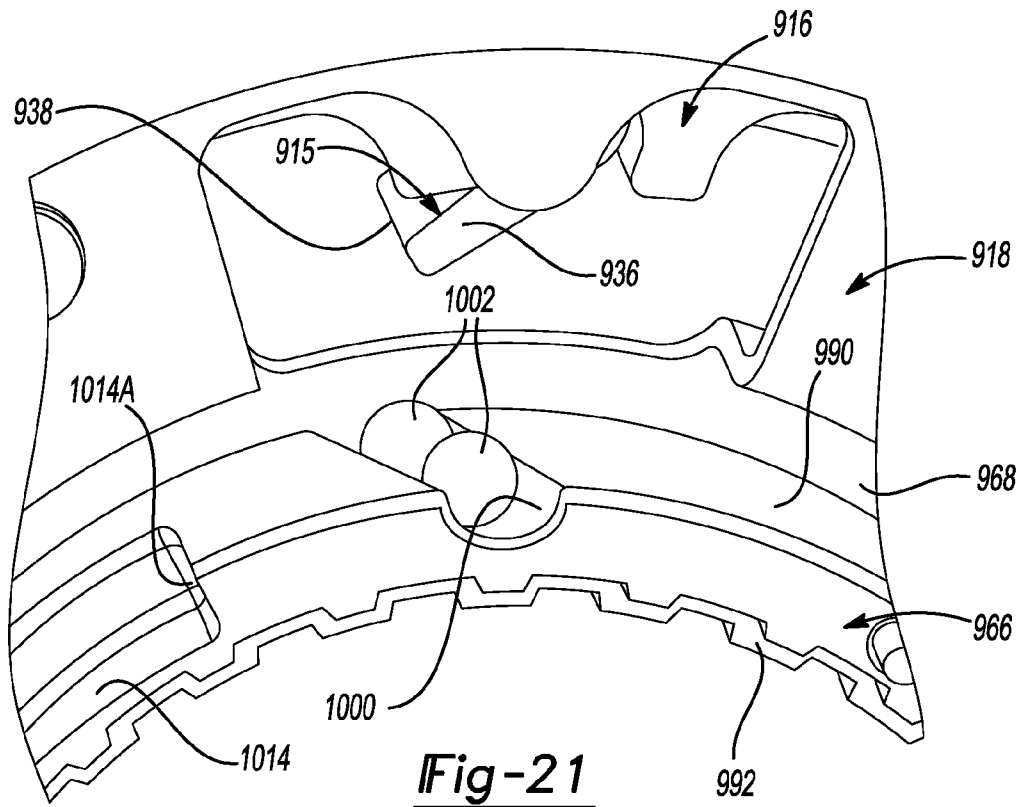


Fig-20



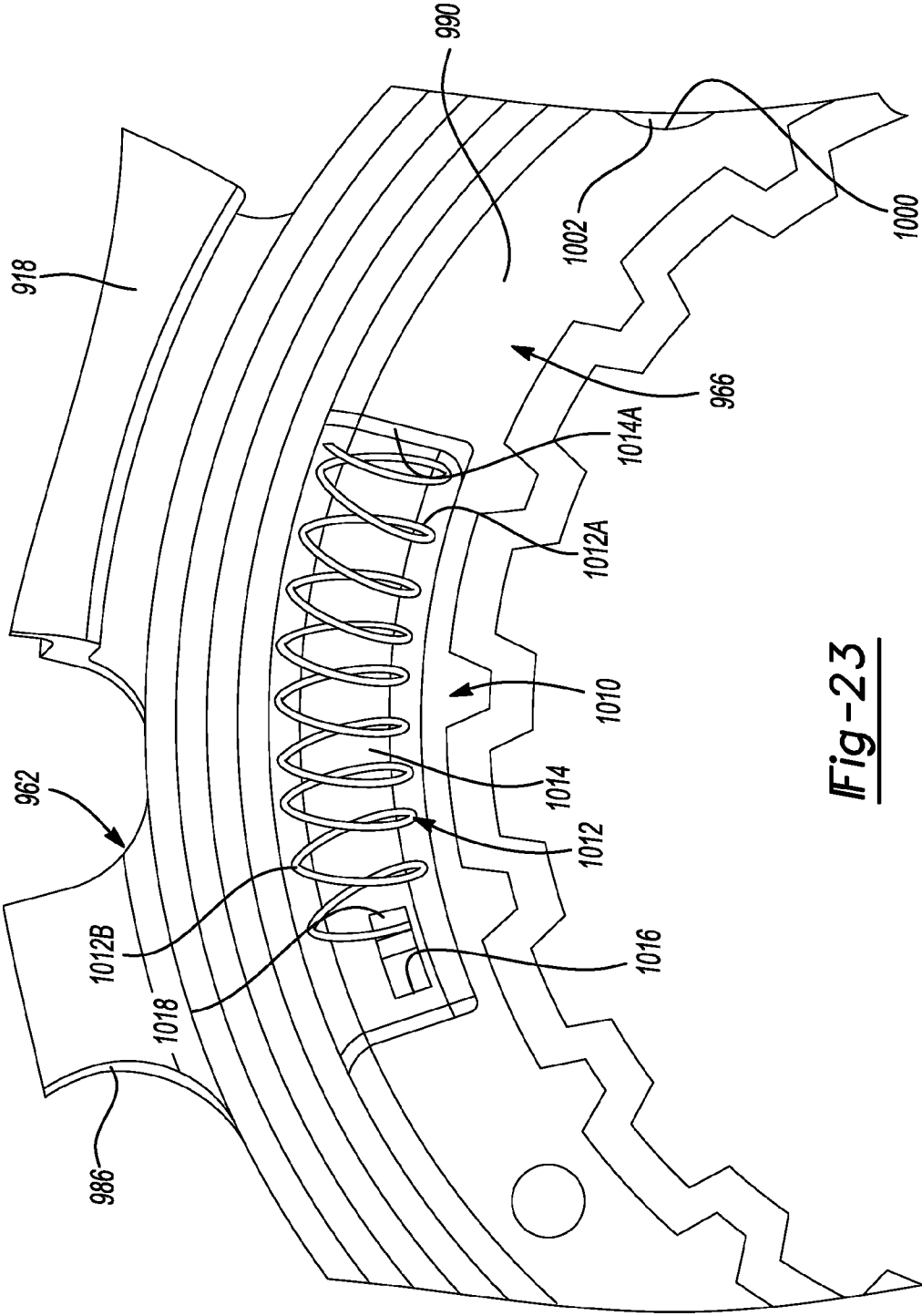


Fig-23



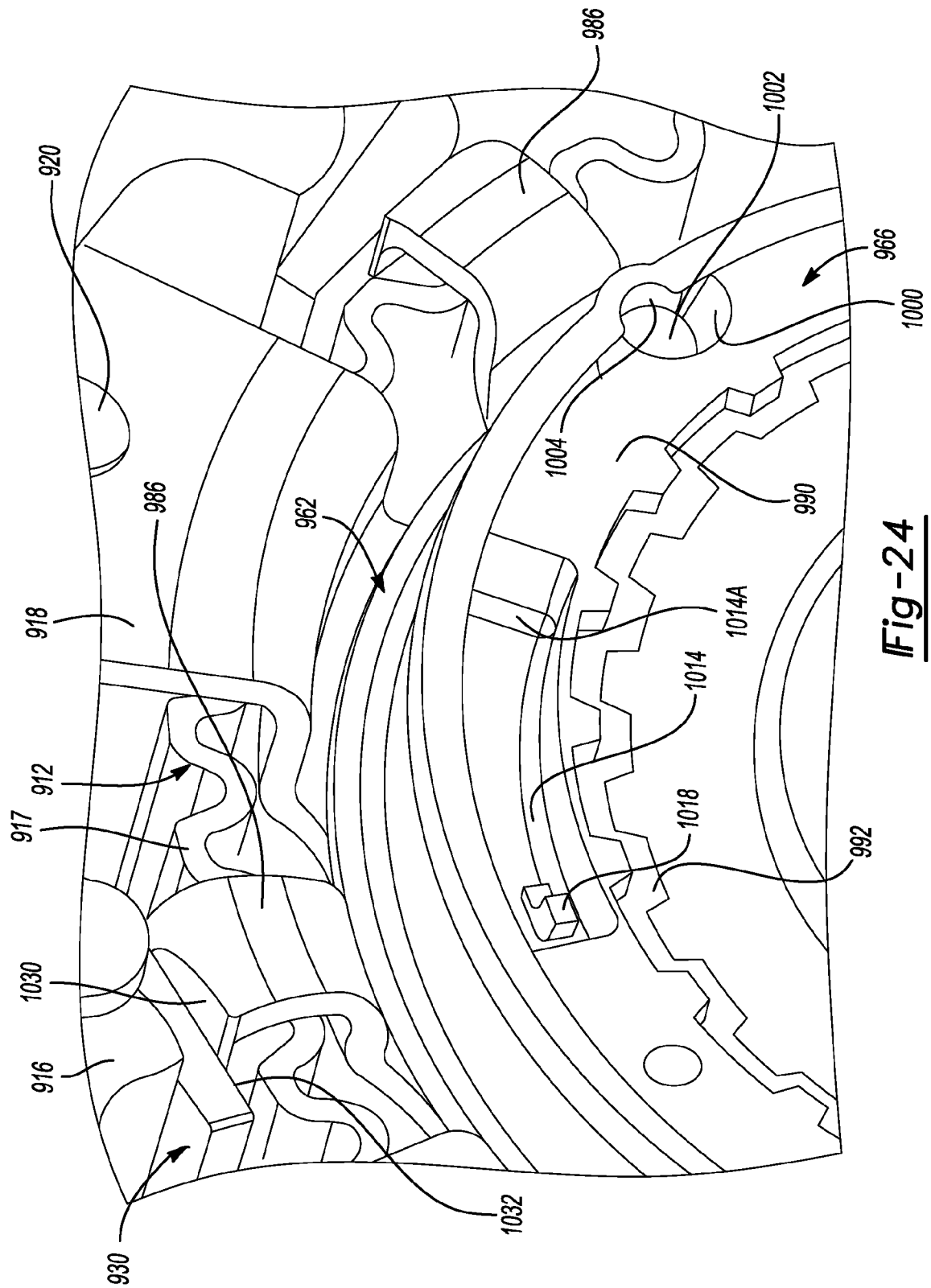


Fig-24

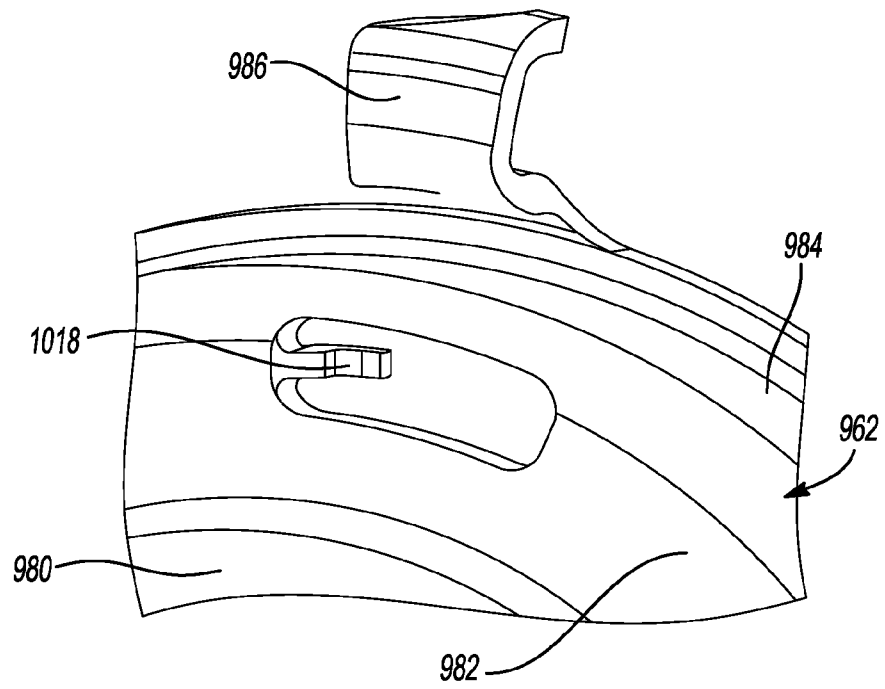


Fig-25

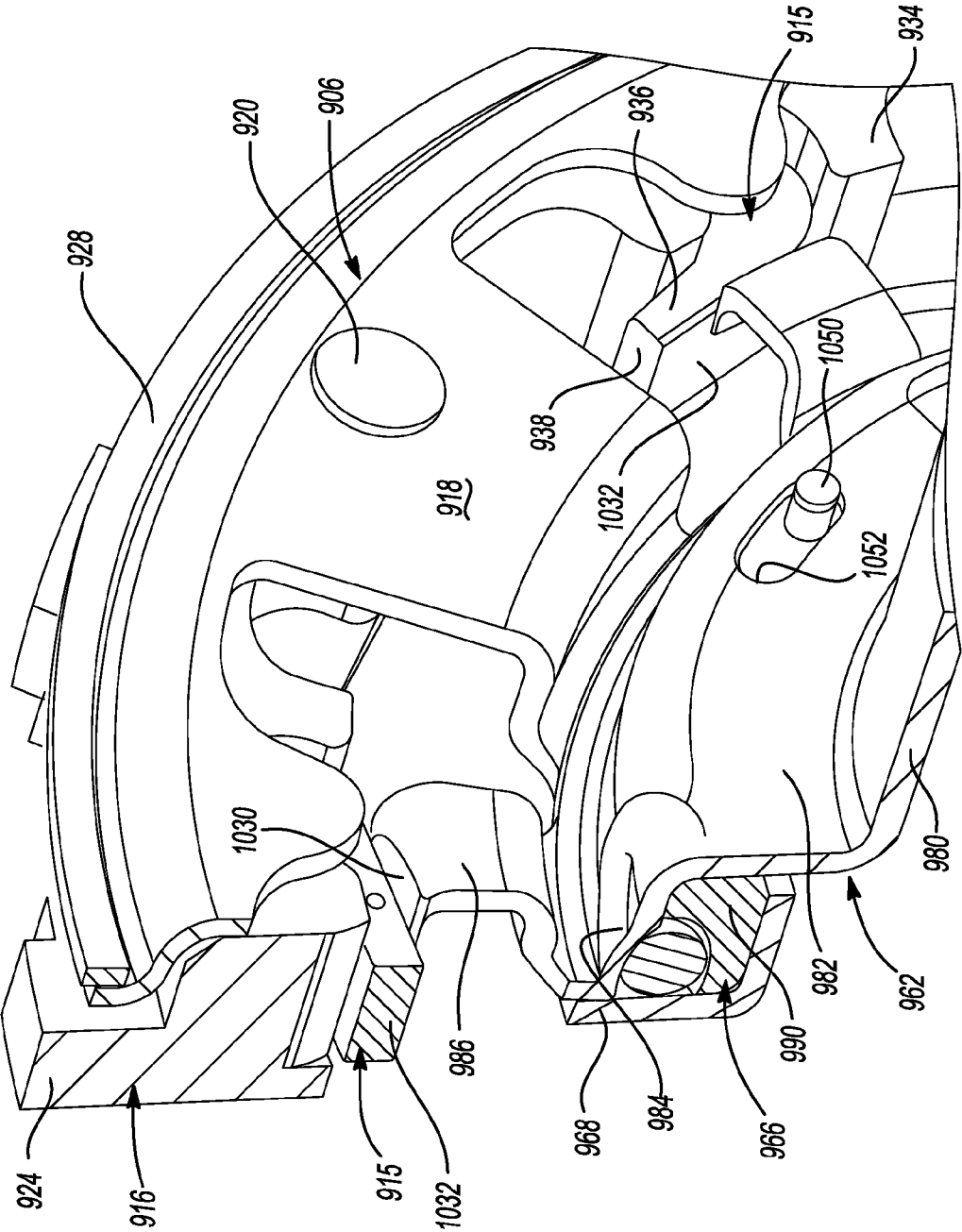


Fig-26

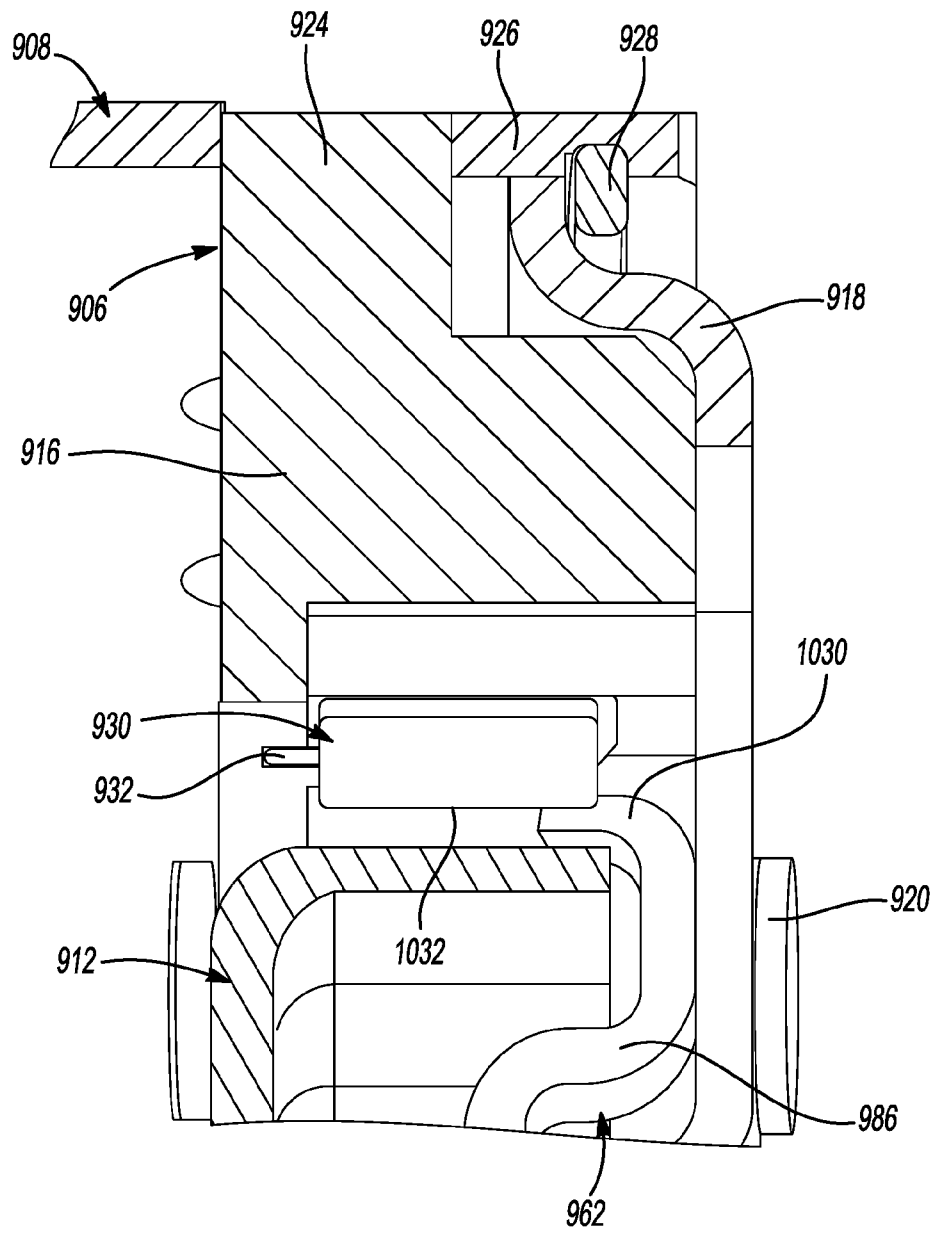


Fig-27

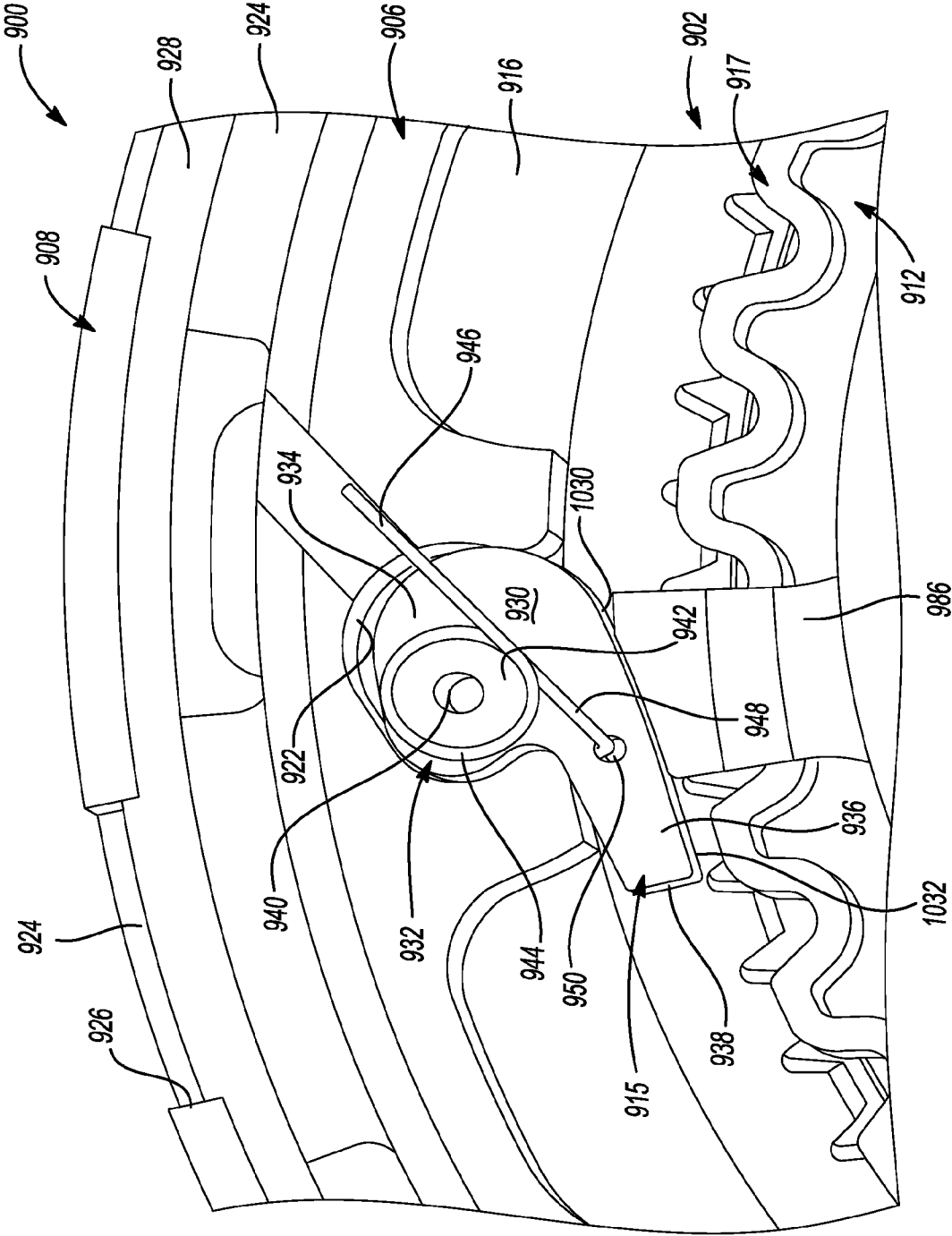


Fig-28

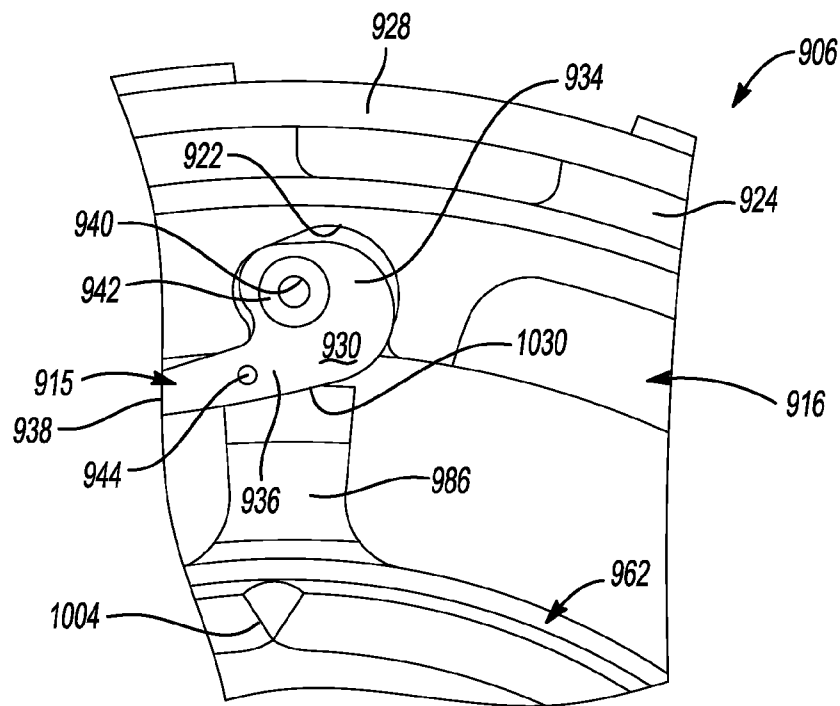


Fig-29

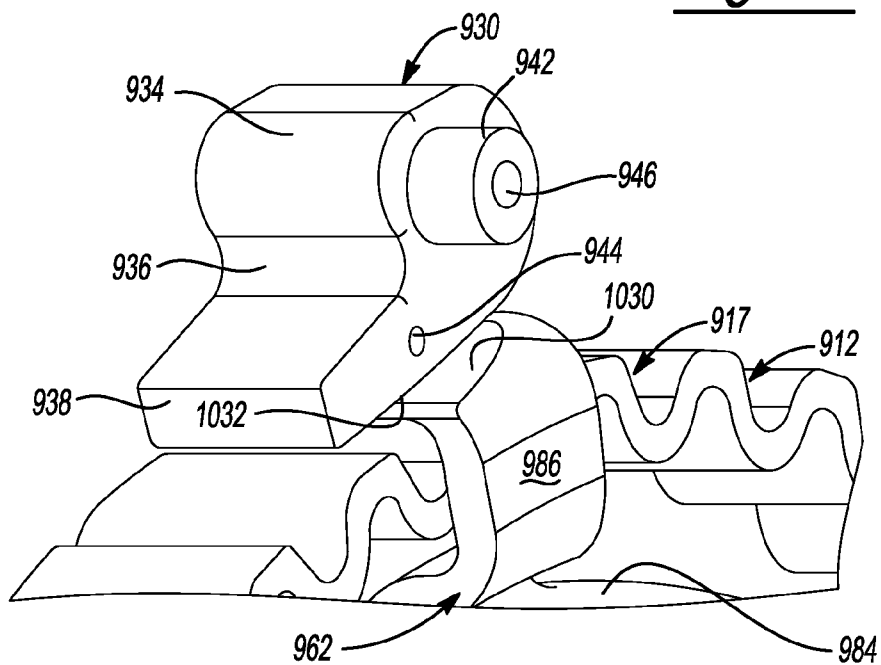


Fig-30

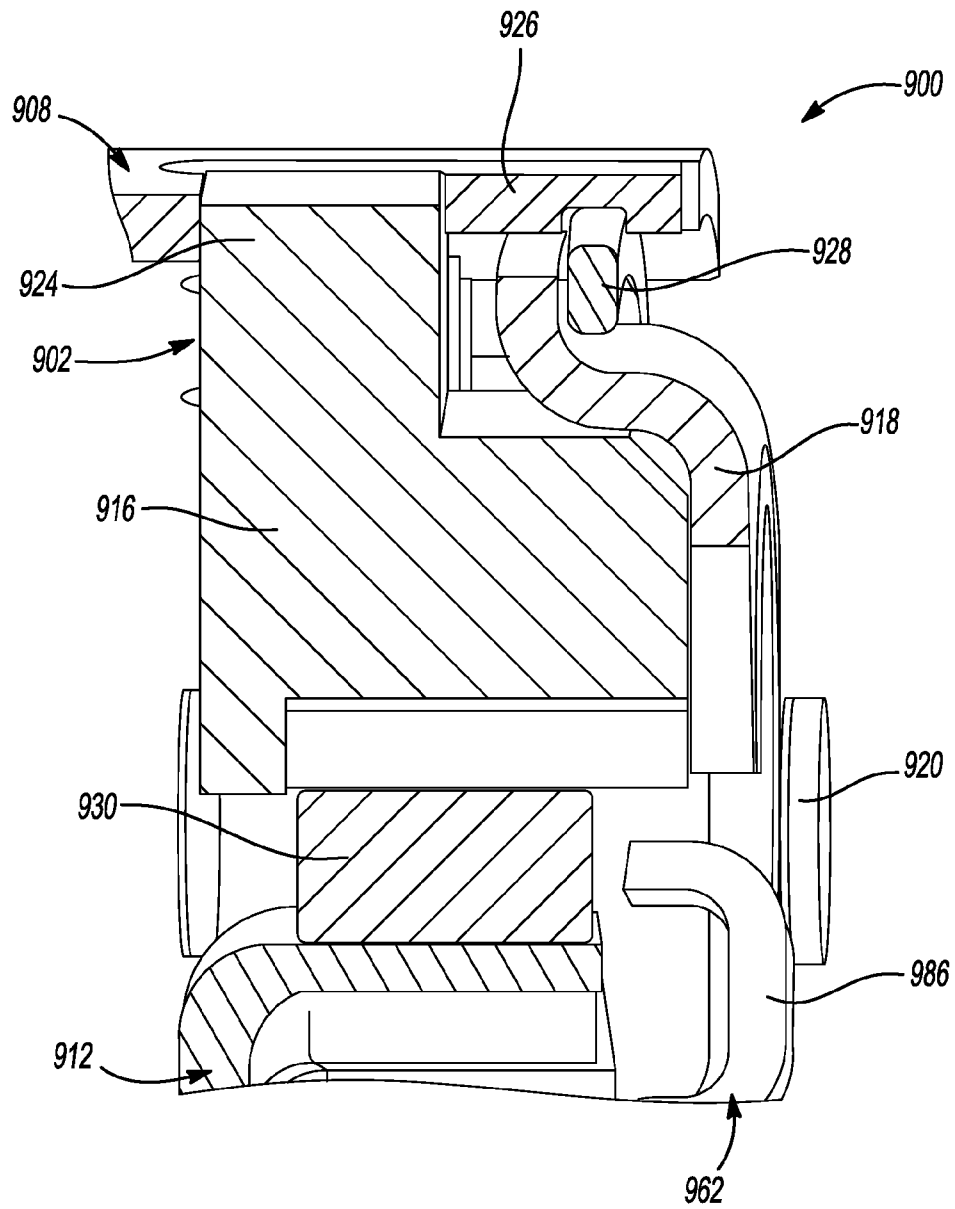


Fig-31

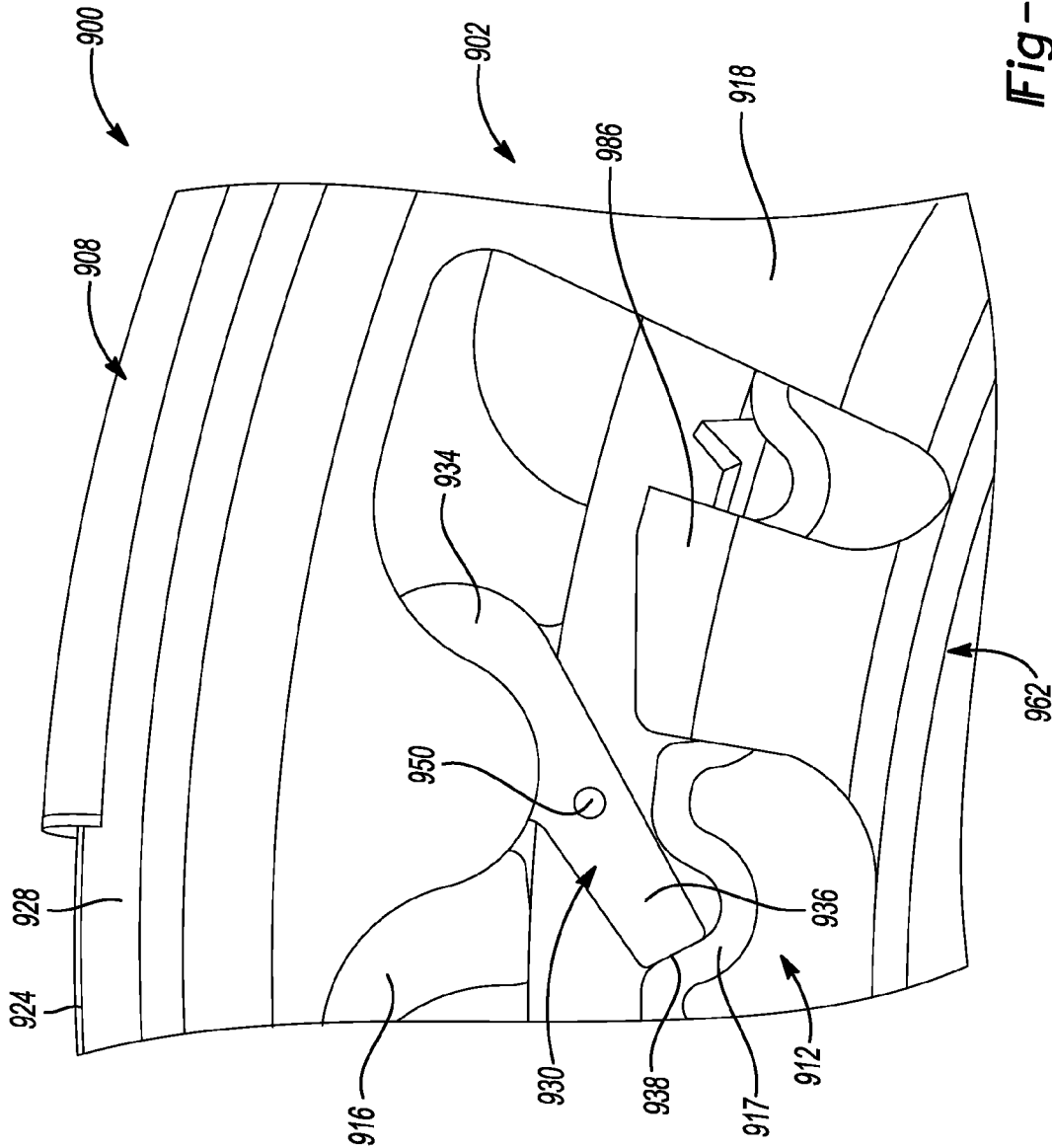


Fig-32

INTERNATIONAL SEARCH REPORT

International application No.

PCT/CA2020/050085

A. CLASSIFICATION OF SUBJECT MATTER

IPC: **F16D 41/08** (2006.01), **B60K 17/02** (2006.01), **F16D 41/16** (2006.01), **F16D 48/06** (2006.01), **F16H 31/00** (2006.01)

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

IPC: **F16D 41/08** (2006.01), **F16D 41/12** (2006.01), **F16D 41/16**

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic database(s) consulted during the international search (name of database(s) and, where practicable, search terms used)

DB: QUESTEL-ORBIT

KW: teeth, ramp

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X Y	US2018266502A1 (SHIORI et al.) 20 September 2018 (20-09-2018)	1, 10 9
Y	US9732809B2 (NIEMIEC et al.) 15 August 2017 (15-08-2017)	9
A	US2018149211A1 (KUROSAKI et al.) 31 May 2018 (31-05-2018)	1-15
A	US10066679B2 (CIOCI et al.) 04 September 2018 (04-09-2018)	1-15
A	US2016356323A1 (ENDO et al.) 08 December 2016 (08-12-2016)	1-15

☐ Further documents are listed in the continuation of Box C.☒ See patent family annex.

* "A" "D" "E" "L" "O" "P"	Special categories of cited documents: document defining the general state of the art which is not considered to be of particular relevance document cited by the applicant in the international application earlier application or patent but published on or after the international filing date document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified) document referring to an oral disclosure, use, exhibition or other means document published prior to the international filing date but later than the priority date claimed	"T" "X" "Y" "&"	later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art document member of the same patent family
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Date of the actual completion of the international search
11 March 2020 (11-03-2020)Date of mailing of the international search report
08 April 2020 (08-04-2020)Name and mailing address of the ISA/CA
Canadian Intellectual Property Office
Place du Portage I, C114 - 1st Floor, Box PCT
50 Victoria Street
Gatineau, Quebec K1A 0C9
Facsimile No.: 819-953-2476

Authorized officer

Alan Jones (819) 639-7677

INTERNATIONAL SEARCH REPORT
Information on patent family members

International application No.
PCT/CA2020/050085

Patent Document Cited in Search Report	Publication Date	Patent Family Member(s)	Publication Date
US2018266502A1	20 September 2018 (20-09-2018)	US10563710B2	18 February 2020 (18-02-2020)
US2018149211A1	31 May 2018 (31-05-2018)	US10247264B2 JP2018084320A JP6540669B2	02 April 2019 (02-04-2019) 31 May 2018 (31-05-2018) 10 July 2019 (10-07-2019)
US9732809B2	15 August 2017 (15-08-2017)	US2016348741A1 CN106195056A CN106195056B DE102016108654A1	01 December 2016 (01-12-2016) 07 December 2016 (07-12-2016) 11 October 2019 (11-10-2019) 01 December 2016 (01-12-2016)
US10066679B2	04 September 2018 (04-09-2018)	US2017138414A1 CN106030143A CN106030143B DE112015001044T5 WO2015127546A1	18 May 2017 (18-05-2017) 12 October 2016 (12-10-2016) 24 April 2018 (24-04-2018) 29 December 2016 (29-12-2016) 03 September 2015 (03-09-2015)
US2016356323A1	08 December 2016 (08-12-2016)	US10197111B2 CN106224405A CN106224405B EP3100887A1 EP3100887B1 JP2016223578A JP6323397B2	05 February 2019 (05-02-2019) 14 December 2016 (14-12-2016) 25 September 2018 (25-09-2018) 07 December 2016 (07-12-2016) 21 November 2018 (21-11-2018) 28 December 2016 (28-12-2016) 16 May 2018 (16-05-2018)