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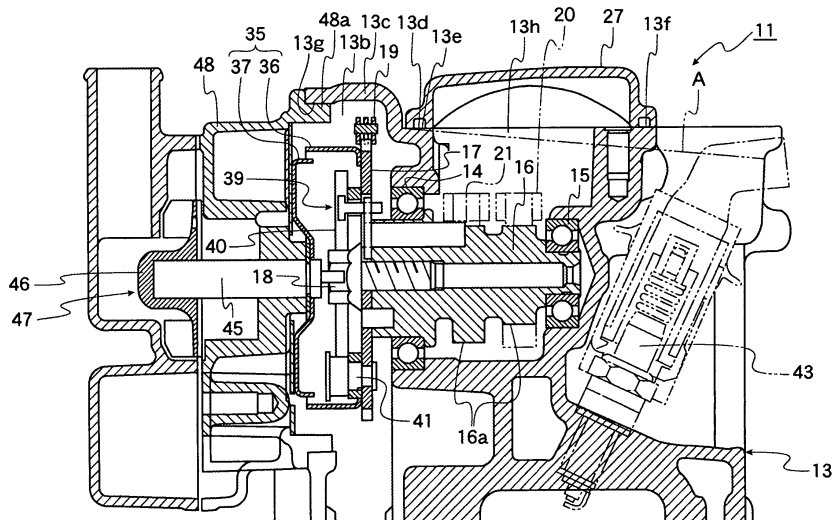
(54) **ENGINE**

(57) It is an object of the invention to provide an engine, which can be decreased in height while the strength of a cylinder head is ensured.

An engine 11 according to the invention is constructed such that a cam chain 19 is trained around a sprocket 17 fixed to a camshaft 16, a roof wall 13c of an accommodation chamber 13b, which accommodates therein

the sprocket 17, is formed integral with a cylinder head 13, an opening 13h is formed on an upper surface of the cylinder head 13 to be adjacent to the roof wall 13c, and intake and exhaust valves 22, 23 face outside from the opening 13h, and that an opened surface of the opening 13h is positioned in a lower position than an upper end of the roof wall 13c and formed on the same plane.

Fig. 1



Description

TECHNICAL FIELD

[0001] This invention relates to an engine, in which a rotating member such as a sprocket, etc. is fixed to a camshaft provided on a side of a cylinder head.

BACKGROUND ART

[0002] Conventionally, with engines, a cylinder head is clamped and fixed to an upper portion of a cylinder block by means of a plurality of clamping bolts, a piston is provided in the cylinder block to be able to reciprocate, an intake valve and an exhaust valve are provided to open and close a combustion chamber, which is defined by the piston, the cylinder block, the cylinder head, and the like, and rocker arms are provided to push the valves.

[0003] Also, a camshaft is provided on the cylinder head to drive the rocker arms, and a sprocket, to which a drive force is transmitted from a crank shaft, is provided on the camshaft. Openings are formed on the cylinder head to have the intake valve, the exhaust valve, and the sprocket exposed outside, and a head cover is provided on an upper portion of the cylinder head to cover the openings. By removing the head cover, tappet adjustment can be made (for example, see JP-A-2000-303906).

[0004] With such conventional engines, however, since large openings are formed on the cylinder head to have the intake valve, the exhaust valve, and the sprocket exposed outside, strength is decreased and in the case where a separate reinforcement plate is mounted above the openings and the head cover is further mounted thereon, the number of parts is increased and an engine is increased in height.

[0005] Also, since openings to have the intake and exhaust valves exposed outside are formed in two locations and inclined in different directions, formation of the openings is enormous.

DISCLOSURE OF THE INVENTION

[0006] Hereupon, the invention has been thought of in view of the above and has its object to provide an engine capable of decreasing a cylinder head in height while ensuring the strength therefor.

[0007] The invention capable of attaining the above object provides an engine, characterized in that a rotating member is fixed to a camshaft provided on a cylinder head, a drive force from a crank shaft is transmitted to the rotating member, and a roof wall of an accommodation chamber, in which the rotating member is accommodated, is formed integral with the cylinder head, and that an opening is formed on an upper surface portion of the cylinder head to be adjacent to the roof wall, intake and exhaust valves are constructed to face outside from the opening, and an opened surface of the opening is posi-

tioned in a lower position than an upper end of the roof wall and on the same plane.

[0008] Also, it is preferred in the engine of the invention that head clamping members for fixation of the cylinder head to a cylinder block be provided inside the opening, and the head clamping members be arranged to face outside through the opening.

[0009] Also, the invention capable of attaining the above object provides an engine, characterized in, that a rotating member is fixed to a camshaft provided on a cylinder head, a wrapping transmission belt for transmission of a drive force from a crank shaft is trained around the rotating member, and an accommodation chamber, which comprises a roof wall and in which the rotating member is accommodated, is formed in the cylinder head, and that an opening is formed on an upper surface portion of the cylinder head to be adjacent to the roof wall, head clamping members for fixation of the cylinder head to a cylinder block are constructed to look out from the opening, and an opened surface of the opening is formed in a lower position than an upper end of the roof wall.

[0010] Also, it is preferred in the engine of the invention that a mating surface on a peripheral portion of the opening, which mates with a head cover for covering the opening, be recessed toward a center of the opening in a position corresponding to a position on a side of the accommodation chamber, in which the rotating member is arranged, whereby an accommodation chamber side recess is formed.

[0011] Also, it is preferred in the engine of the invention that an opposite side of a mating surface on a peripheral portion of the opening, which mates with a head cover for covering the opening, to the accommodation chamber be recessed toward a center of the opening to form a plug-side recess, and a spark plug be arranged in the plug-side recess.

[0012] Also, it is preferred in the engine of the invention that the intake and exhaust valves face outside from the single opening.

[0013] Also, it is preferred in the engine of the invention that the opened surface of the opening be inclined relative to an axis of a cylinder.

[0014] Also, it is preferred in the engine of the invention that the roof wall be arcuate in shape as viewed along the camshaft to have a larger diameter than that of the rotating member.

BRIEF DESCRIPTION OF THE DRAWINGS

[0015]

Fig. 1 is a cross sectional view showing a cylinder head according to an embodiment of the invention. Fig. 2 is a partially broken, front view showing the cylinder head according to the embodiment. Fig. 3 is a view showing the embodiment on a right side of Fig. 2.

Fig. 4 is a partially broken, front view showing a state, in which an AIS according to the embodiment is arranged.

Fig. 5 is a cross sectional view showing intake and exhaust valves according to the embodiment.

Fig. 6 is a plan view showing the cylinder head according to the embodiment.

Fig. 7 is a side view showing the cylinder head according to the embodiment.

[0016] In addition, the reference numeral 11 in the drawings denotes an engine, 13 a cylinder head, 13a insertion holes, 13b an accommodation chamber, 13c a roof wall, 13d a mating surface, 13e an accommodation chamber side recess, 13f a plug-side recess, 13g an accommodation-chamber opening, 13h an opening, 16 a camshaft, 17 a sprocket (rotating member), 19 a cam chain (wrapping transmission belt), 22 intake valves, 23 exhaust valves, 27 a head cover, 31 stud bolts (head clamping members), and 32 clamping bolts (head clamping members).

BEST MODE FOR CARRYING OUT THE INVENTION

[0017] An embodiment of the invention will be described below.

[0018] Figs. 1 to 7 show the embodiment of the invention.

[0019] First, the constitution is described, and the reference numeral 11 in Fig. 1 denotes a single cylinder, four-cycle, four-valve engine provided on a motorcycle, the engine 11 comprising a cylinder head 13 fixed to a cylinder block (illustration of which is omitted).

[0020] As shown in Fig. 1, a camshaft 16 is rotatably provided on the cylinder head 13 through bearings 14, 15, which are different in diameter from each other, and a sprocket 17 as a "rotating member" is mounted to an end of the camshaft 16 by means of a bolt 18. A cam chain 19 as a "wrapping transmission belt" is trained around the sprocket 17, so that a drive force from a crank shaft is transmitted to the sprocket 17 and the camshaft 16 through the cam chain 19.

[0021] The camshaft 16 is formed with intake and exhaust cam portions 16a, and one ends 20a, 21a of an intake rocker arm 20 and an exhaust rocker arm 21 are caused to abut against the cam portions 16a while the other ends 20b, 21b are caused to abut against intake valves 22 and exhaust valves 23, as shown in Fig. 5. Thus, when the camshaft 16 is rotated, the respective rocker arms 20, 21 swing on turning shafts 20c, 21c, so that the respective valves 22, 23 are actuated in a predetermined timing to open and close an intake passage 24 and an exhaust valve 25.

[0022] The intake valves 22 and the exhaust valves 23, respectively, are provided two, that is, four in total, an opening 13h is formed above the intake valves 22 and the exhaust valves 23, and a head cover 27 as a "cover" is detachably mounted on the opening 13h by means of

mount screws 28 (see Figs. 1 and 6).

[0023] Also, stud bolts 31 and clamping bolts 32, which serve as "head clamping members", respectively, are arranged in two locations on both sides of the camshaft 16 in the opening 13h as shown in Fig. 6, so that the cylinder head 13 is mounted to a cylinder block by means of the stud bolts 31 and the clamping bolts 32.

[0024] More specifically, the stud bolts 31 as "clamping bolts" are inserted into and fixed to insertion holes formed on the cylinder block whereby the stud bolts 31 are provided to project from the cylinder block. The stud bolts 31 are inserted into insertion holes 13a of the cylinder head 13 to be arranged in parallel to an axis of a cylinder. The clamping nuts 32 are screwed onto the stud bolts 31 whereby the cylinder head 13 is mounted on the cylinder block.

[0025] By removing the head cover 27, the clamping nuts 32 are exposed and can be attached or removed, and by removing the clamping nuts 32, the cylinder head 13 can be removed from the cylinder block.

[0026] On the other hand, as shown in Fig. 1, a breather rotating body 36 of a breather device 35 is mounted to the sprocket 17 fixed to a left end of the camshaft 16 in the figure, and a breather stationary body 37 is provided in opposition to the breather rotating body 36. Also, a turning member 40 of a decompression device 39 is arranged in the breather rotating body 36 to be able to turn on a turning shaft 41, and the turning member 40 is turned due to centrifugal forces generated when the sprocket 17 is rotated.

[0027] The sprocket 17, the breather device 35, etc. are accommodated in an accommodation chamber 13b of the cylinder head 13, and a roof wall 13c of the accommodation chamber 13b is formed integral with the cylinder head 13. The accommodation chamber 13b is formed adjacent to the opening 13h, an opened surface of the opening 13h is set to be in a lower position than an upper end of the roof wall 13c, and the roof wall 13c is arcuate in shape as viewed along the camshaft 16 to have a larger diameter than that of the sprocket 17.

[0028] A mating surface 13d on a peripheral portion of the opening 13h, which mates with the head cover 27 for opening and closing of the opening 13h, is formed with an accommodation chamber side recess 13e positioned to correspond to a position on a side of the accommodation chamber 13b, in which the sprocket 17 is arranged, the recess being recessed toward a center of the opening. The studbolts 31 and the clamping nuts 32 are arranged in positions on both sides of the accommodation chamber side recess 13e.

[0029] Also, a plug-side recess 13f is formed on a side of the mating surface 13d opposed to the accommodation chamber 13b to be recessed toward a center of the opening, and a spark plug 43 is arranged in the plug-side recess 13f.

[0030] Further, a mount surface 13i is formed integrally on the mating surface 13d as shown in Figs. 4 and 6, and an AIS (air induction system) 50 is mounted on the mount

surface 13i.

[0031] Moreover, an accommodation-chamber opening 13g is formed laterally of the accommodation chamber 13b, and a fitting portion 48a of a water-pump housing 48 of a water pump device 47 is fitted into and mounted to the accommodation-chamber opening 13g in a sealed state. The water pump device 47 comprises an impeller shaft 45 and an impeller 46, and the impeller shaft 45 is fitted into the bolt 18 mounted to the camshaft 16 whereby the impeller 46 is rotated as the camshaft 16 rotates.

[0032] With such construction, the cylinder head 13 can be enhanced in strength by forming the roof wall 13c of the accommodation chamber 13b, in which the sprocket 17 is accommodated, integral with the cylinder head 13. Besides, since it is possible to ensure stiffness for the cylinder head 13 as described above, the cylinder head 13 can be decreased in height by forming the opened surface of the opening 13h in a lower position than an upper end of the roof wall 13c. That is, when the opening 13h is formed contiguous to the accommodation chamber 13b, stiffness is decreased, so that there is a need of providing a separate reinforcement plate on the upper surface of the opening 13h and when the head cover 27 is fixed thereon, the cylinder head 13 will be increased in height. In contrast, according to the invention, since the roof wall 13c is integrally formed to enhance the stiffness of the cylinder head 13, there is no need of providing any separate reinforcement plate, and since the opening 13h is formed in a lower position than the roof wall 13c, the cylinder head 13 can be decreased in height even in the case where the head cover 27 is mounted on the opening 13h.

[0033] Also, since the accommodation chamber 13b is formed with the accommodation-chamber opening 13g laterally of the sprocket 17 and even in the case where the accommodation-chamber opening 13g is adapted to be closed by the water-pump housing (closure member) 48, the roof wall 13c is formed integral with the cylinder head 13, a peripheral edge (mating surface) of the accommodation-chamber opening 13g will not be shifted midway but can surely come into contact with a mating surface of the fitting portion 48a of the water-pump housing 48 to ensure the sealing quality.

[0034] That is, if the roof wall 13c were separate from the cylinder head 13, the peripheral edge (sealed portion) of the accommodation-chamber opening 13g would become discontinuous, so that there is a fear that shift, poor sealing, or the like is generated between the sealed portion and the roof wall 13c due to error in molding, assembling, or the like. In contrast, according to the invention, the roof wall 13c is formed integral with the cylinder head 13 to eliminate such fear of generation of poor sealing, or the like.

[0035] Further, since the opened surface of the opening 13h is formed on the same plane, it can be formed more easily than the case where a plurality of openings having different inclinations should be formed.

[0036] Moreover, the stud bolts 31 can be made short

and stable in axial forces by arranging the stud bolts 31 and the clamping nuts 32 inside the opening 13h, and since the stud bolts 31 and the clamping nuts 32 do not project above the head cover 27, the engine 11 can be decreased in height. Also, by removing the head cover 27, it is possible to perform tappet adjustment and to attach or remove the stud bolts 31 and the clamping nuts 32, thus enabling attaching or removing the cylinder head 13 from the cylinder block.

[0037] Also, the mating surface 13d on the peripheral edge of the opening 13h is recessed toward the center of the opening 13h in a position corresponding to a position, in which the sprocket 17 on the side of the accommodation chamber 13b is arranged, to form the accommodation chamber side recess 13e, so that the sprocket 17 can be made close to the respective valves 22, 23 to make the cylinder head 13 (an upper portion of the engine 11) further compact in width along the camshaft 16.

[0038] Further, by arranging the spark plug 43 in the plug-side recess 13f about the mating surface 13d on the peripheral edge of the opening 13h, the spark plug 43 can be arranged between the respective valves 22, 23 to stand further upright, so that the cylinder head 13 can be made further compact in width along the camshaft 16.

[0039] Moreover, since the intake valves 22 and the exhaust valves 23 face outside from inside the single opening 13h, adjustment of both the intake valves 22 and the exhaust valves 23 can be made by removing the head cover 27, which covers the opening 13h.

[0040] Besides, since the roof wall 13c is arcuate in shape as viewed along the camshaft 16 to have a larger diameter than that of the sprocket 17, a useless space in the roof wall 13c can be made as small as possible in a state, in which the sprocket 17 is arranged in the roof wall 13c, so that it is possible to make the roof wall 13c as compact as possible in external shape.

[0041] Also, as shown in Figs. 4 and 6, the mating surface 13d is formed integral with the mount surface 13i, to which the AIS (air induction system) 50 is mounted, so that the mount surface 13i can be easily formed, and by arranging the AIS 50 on the head cover 27, the AIS 50 can be simply arranged only by mounting the head cover 27 on the mating surface 13d.

[0042] By the way, the opened surface of the opening 13h can be inclined relative to an axis of the cylinder as indicated by a two-dot chain line A in Fig. 1 in a manner to be lowered on an opposite side to the roof wall 13c. With such arrangement, a tool is inserted from the opening 13h to enable performing the attaching or removing operation from laterally of the stud bolts 31 and the clamping nuts 32, and so the operation of maintenance is easy to perform.

[0043] In addition, while the embodiment applies the invention to the engine 11 for motorcycles, the invention is of course applicable to engines for other uses. Also, engines are not limited to single cylinder, four-valve ones but may be of other type. Further, while the sprocket 17 is adopted as a "rotating member", it is not limitative but

it is possible to use a pulley in place of the sprocket 17 and a belt as a "wrapping transmission belt" in place of the cam chain 19. Also, it is possible to adopt gears in place of the sprocket 17 so that a drive force from the crank shaft can be transmitted by means of a plurality of gears.

[0044] While the invention has been described in detail with reference to the specific embodiment, it is apparent to those skilled in the art that various changes and modifications can be made without departing from the spirit and scope of the invention.

[0045] The present application is based on Japanese Patent Application (Patent Application No. 2003-195211) filed on July 10, 2003 and Japanese Patent Application (Patent Application No. 2004-176406) filed on June 15, 2004, contents of which are incorporated herein for reference.

INDUSTRIAL APPLICABILITY

[0046] According to the invention, the cylinder head can be enhanced in strength by forming the roof wall of the accommodation chamber, in which the rotating member is accommodated, integral with the cylinder head. Besides, since it is possible to ensure stiffness for the cylinder head as described above, the cylinder head can be decreased in height by forming the opened surface of the opening in a lower position than an upper end of the roof wall. That is, when the opening is formed contiguous to an upper portion of the accommodation chamber without forming the roof wall integral with the cylinder head, stiffness is decreased, so that there is a need of providing a separate reinforcement plate on the upper surface of the opening and since the head cover is fixed thereon, the cylinder head will be increased in height. In contrast, according to the invention, since the roof wall is integrally formed to enhance the stiffness of the cylinder head, there is no need of providing any separate reinforcement plate, and since the opening is formed in a lower position than the roof wall, the cylinder head can be decreased in height even in the case where the head cover is mounted on the opening.

[0047] Also, since the accommodation chamber is formed with the accommodation-chamber opening laterally of the rotating member and even in the case where the accommodation-chamber opening is adapted to be closed by the closure member, the roof wall is formed integral with the cylinder head, a peripheral edge (mating surface) of the accommodation-chamber opening will not be shifted midway but can be caused to surely abut against a mating surface of the closure member to ensure the sealing quality.

[0048] Further, since the opened surface is formed on the same plane, it can be formed more easily than the case where a plurality of openings having different inclinations should be formed.

[0049] Moreover, according to the invention, the head clamping members are arranged inside the opening

whereby the head clamping members can be positionally lowered, so that the head clamping members (for example, clamping bolts) can be made short and stable in axial forces, and since the head clamping members do not project above the head cover, the engine can be decreased in height. Also, by removing the head cover, it is possible to perform tappet adjustment and to attach or remove the head clamping members, thus enabling attaching or removing the cylinder head from the cylinder block.

[0050] Also, the accommodation chamber side recess is formed on the mating surface on the peripheral edge of the opening to be positioned in a position corresponding to a position, in which the rotating member on the side of the accommodation chamber is arranged, so that the rotating member can be made close to the valves to make the cylinder head further compact in width along the camshaft.

[0051] Also, by arranging the spark plug in the plug-side recess about the mating surface on the peripheral edge of the opening, which recess is recessed toward the center of the opening, the spark plug can be arranged between the respective valves to stand further upright, so that the cylinder head can be made further compact in width along the camshaft.

[0052] Also, since the intake valves and the exhaust valves face outside from inside the single opening, adjustment of both the intake valves and the exhaust valves can be made by removing the head cover, which covers the opening.

[0053] Also, since the opened surface of the opening is inclined, a tool is inserted from the opening to enable performing the attaching or removing operation from laterally of the head clamping members, and so maintenance is easy to perform.

[0054] Also, since the roof wall is formed to be arcuate corresponding to the circular-shaped rotating member, a useless space in the roof wall can be made as small as possible in a state, in which the rotating member is arranged in the roof wall, it is possible to make the roof wall as compact as possible in external shape.

Claims

1. An engine, **characterized in that** a rotating member is fixed to a camshaft provided on a cylinder head, a drive force from a crank shaft is transmitted to the rotating member, and a roof wall of an accommodation chamber, in which the rotating member is accommodated, is formed integral with the cylinder head, and that an opening is formed on an upper surface portion of the cylinder head to be adjacent to the roof wall, intake and exhaust valves are constructed to face outside from the opening, and an opened surface of the opening is positioned in a lower position than an upper end of the roof wall and on the same plane.

2. The engine according to claim 1, wherein head clamping members for fixation of the cylinder head to a cylinder block are provided inside the opening, and the head clamping members are arranged to face outside through the opening. 5

3. An engine, **characterized in that** a rotating member is fixed to a camshaft provided on a cylinder head, a wrapping transmission belt for transmission of a drive force from a crank shaft is trained around the rotating member, and an accommodation chamber, which comprises a roof wall and in which the rotating member is accommodated, is formed in the cylinder head, and 10
 that an opening is formed on an upper surface portion of the cylinder head to be adjacent to the roof wall, head clamping members for fixation of the cylinder head to a cylinder block are constructed to look out from the opening, and an opened surface of the opening is formed in a lower position than an upper end of the roof wall. 15 20

4. The engine according to any one of claims 1 to 3, wherein a mating surface on a peripheral portion of the opening, which mates with a head cover for covering the opening, is recessed toward a center of the opening in a position corresponding to a position on a side of the accommodation chamber, in which the rotating member is arranged, whereby an accommodation chamber side recess is formed. 25 30

5. The engine according to any one of claims 1 to 3, wherein an opposite side of a mating surface on a peripheral portion of the opening, which mates with a head cover for covering the opening, to the accommodation chamber is recessed toward a center of the opening to form a plug-side recess, and a spark plug is arranged in the plug-side recess. 35

6. The engine according to any one of claims 1 to 3, wherein the intake and exhaust valves face outside from the single opening. 40

7. The engine according to any one of claims 1 to 3, wherein the opened surface of the opening is inclined relative to an axis of a cylinder. 45

8. The engine according to any one of claims 1 to 3, wherein the roof wall is arcuate in shape as viewed along the camshaft to have a larger diameter than that of the rotating member. 50

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Fig. 1

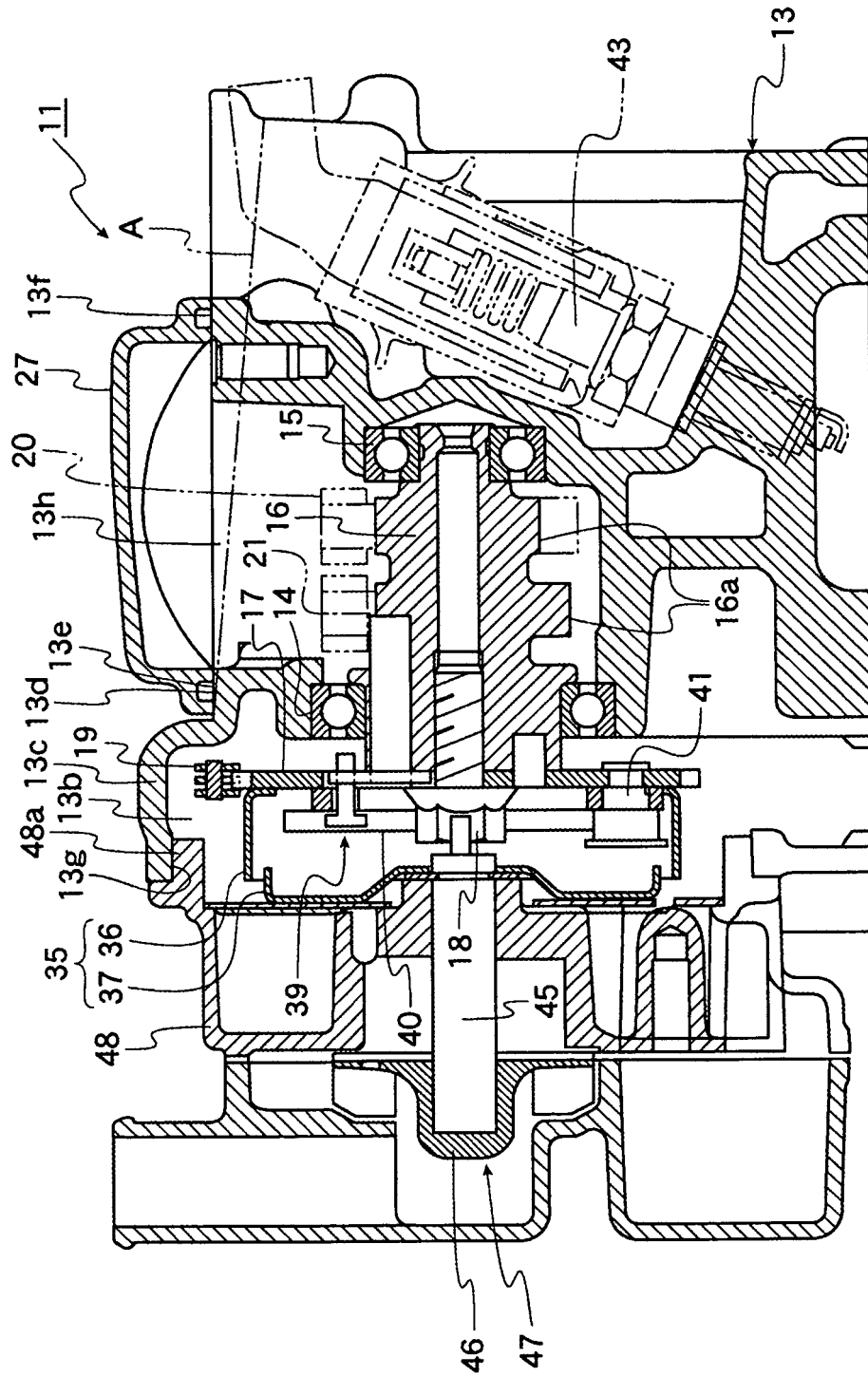
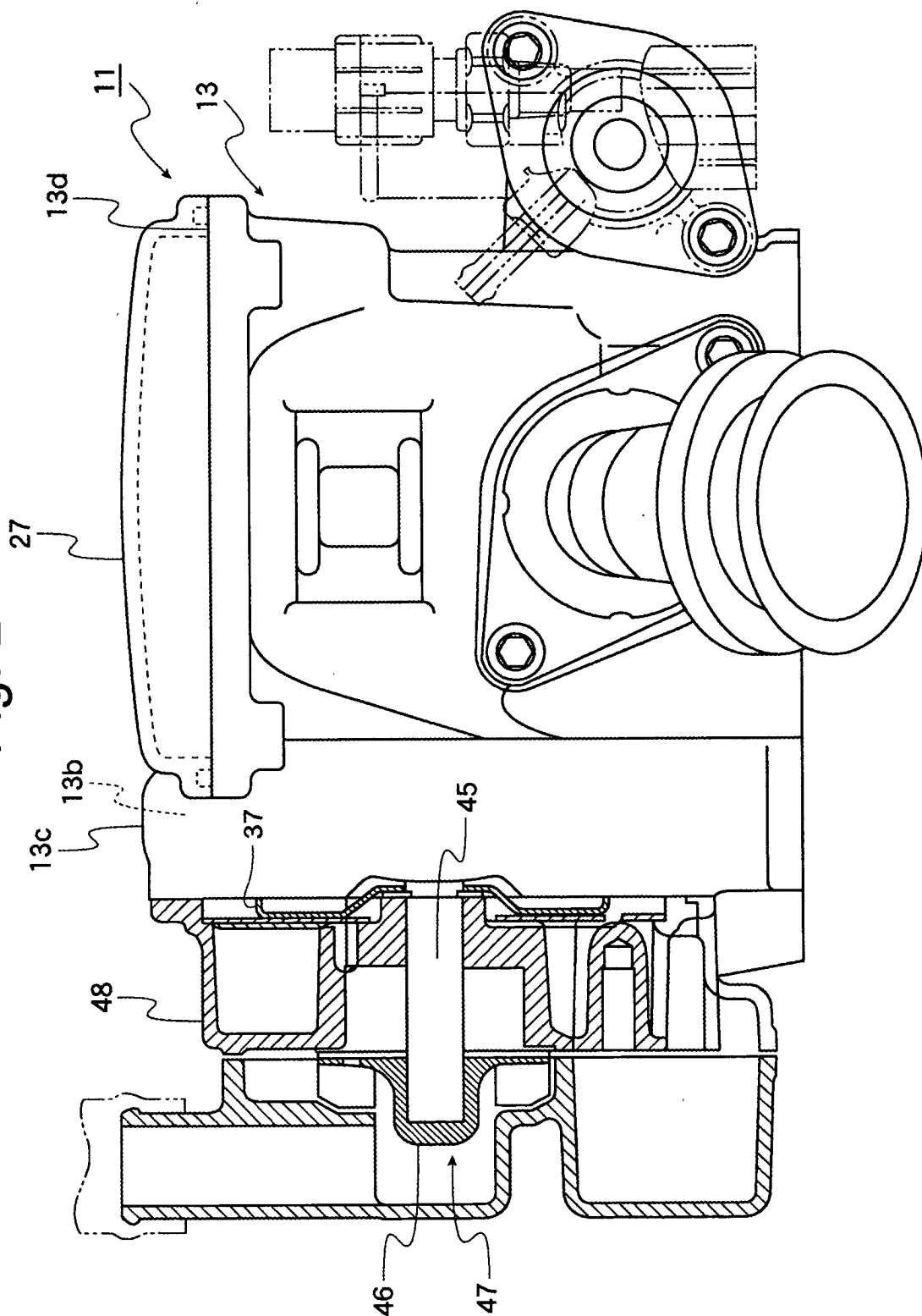


Fig. 2



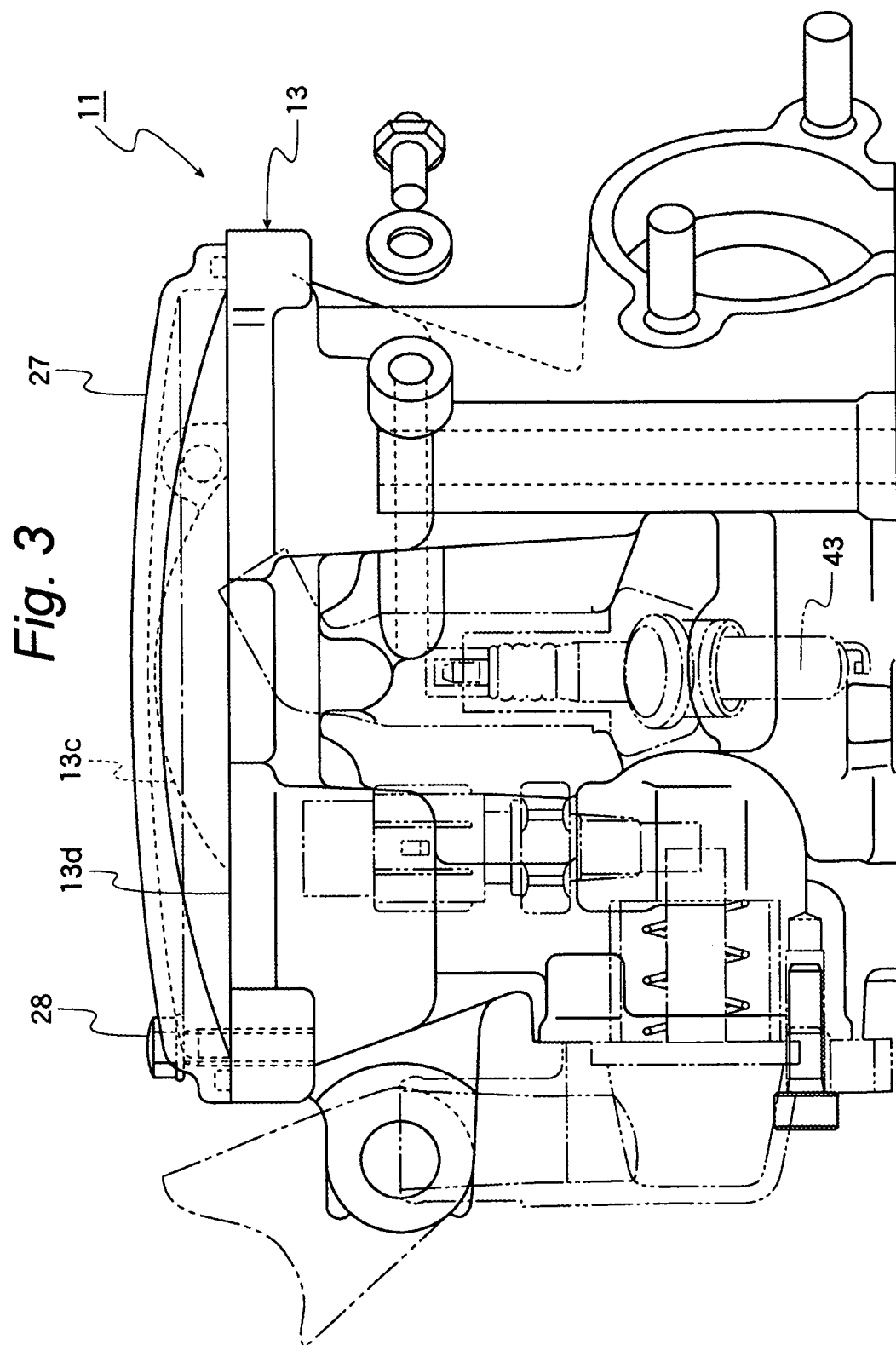


Fig. 4

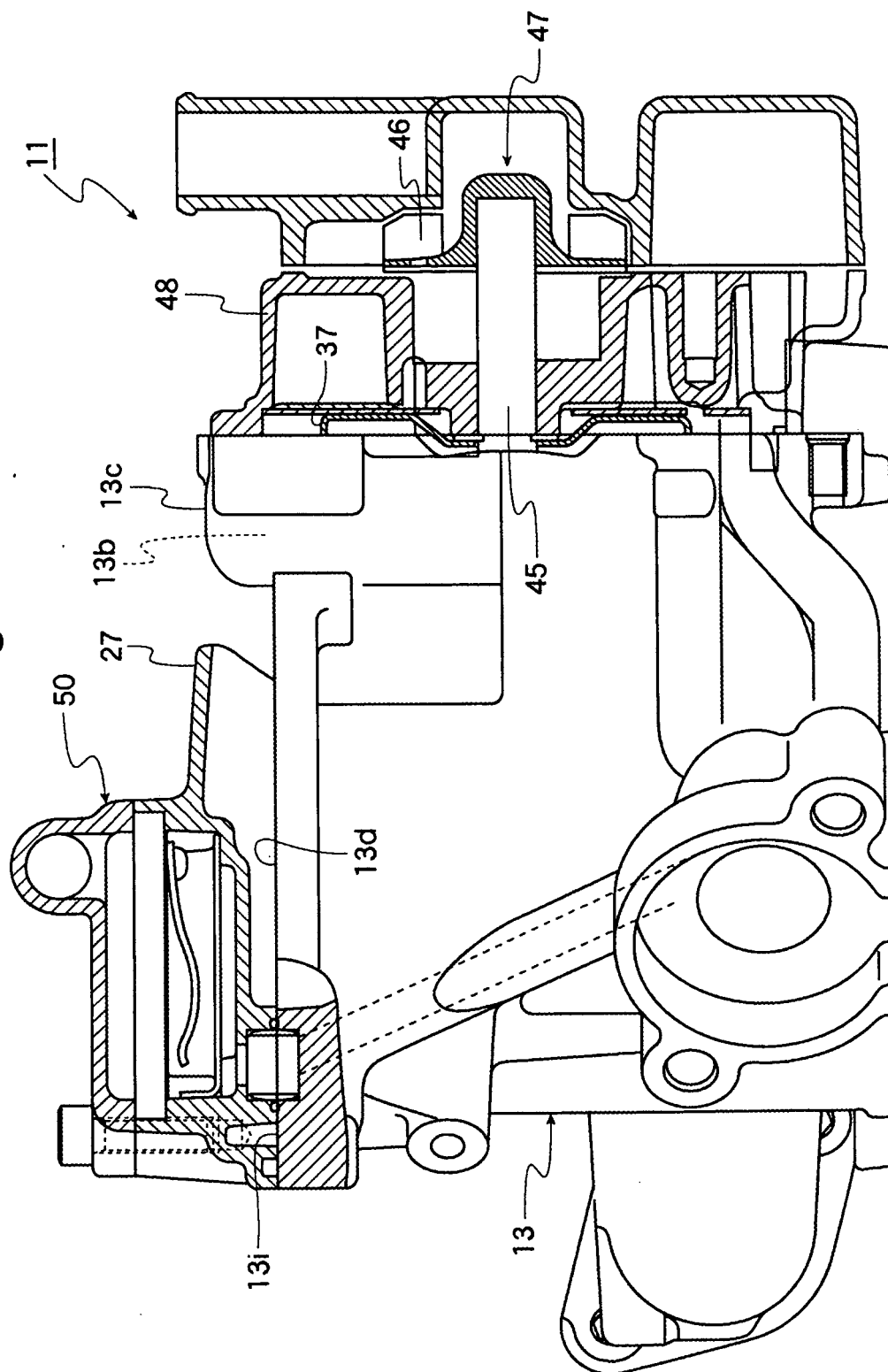


Fig. 5

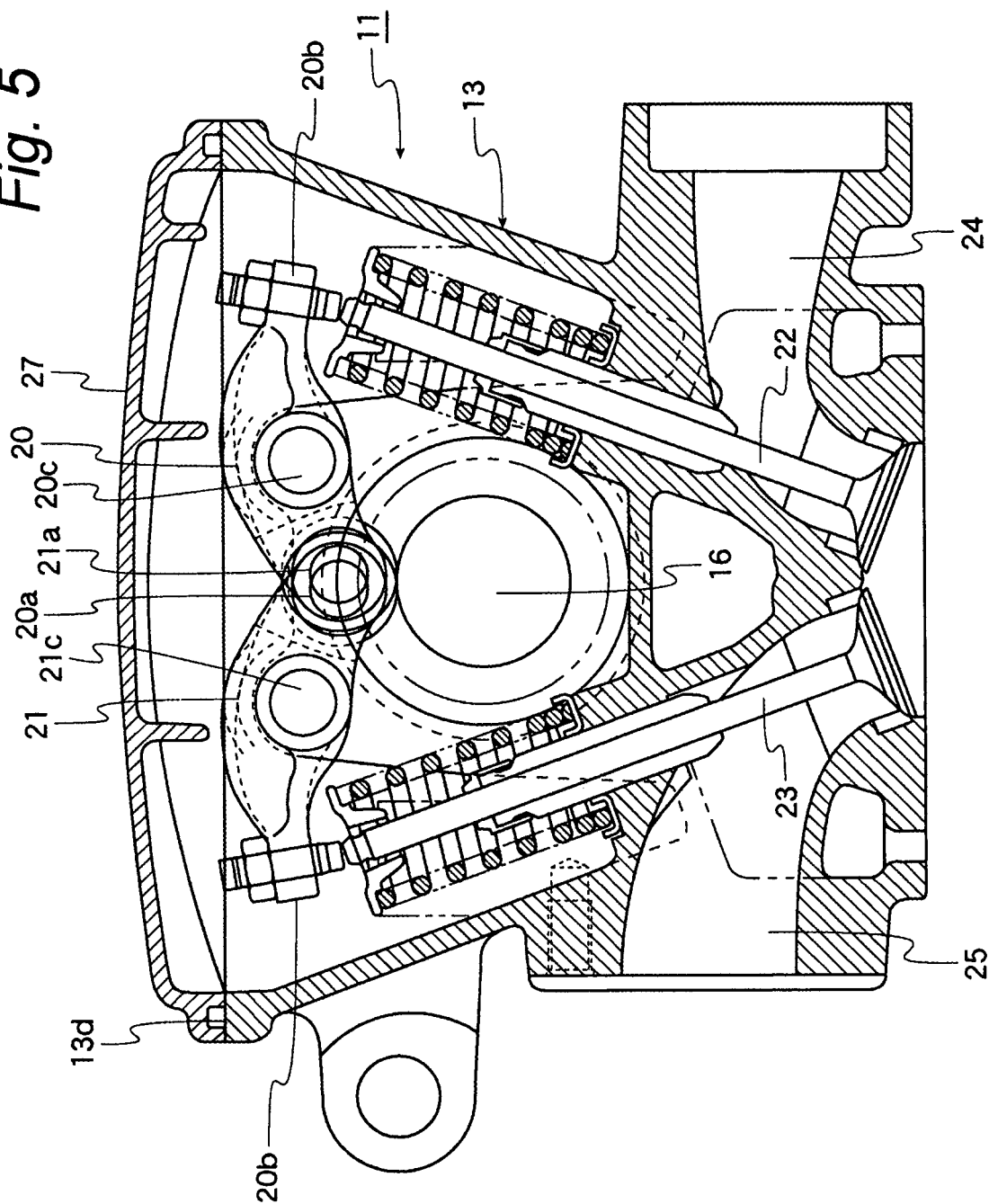
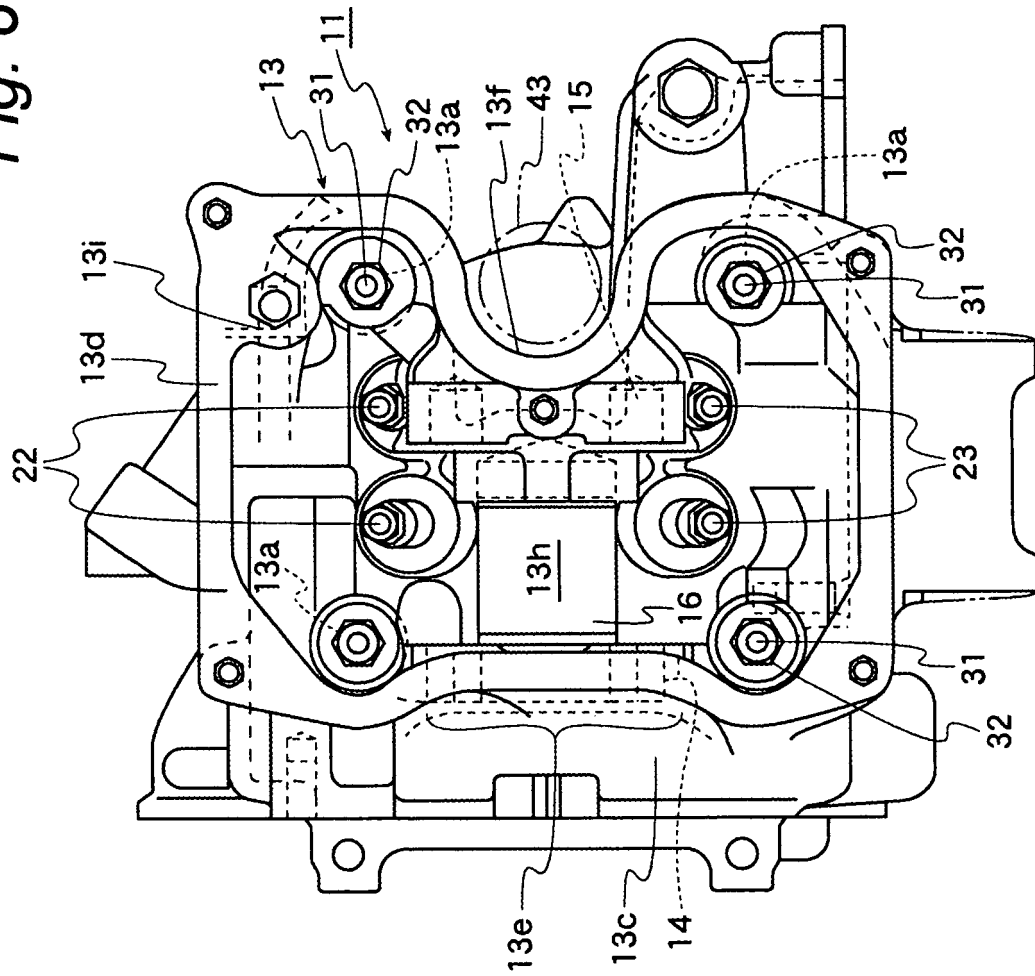
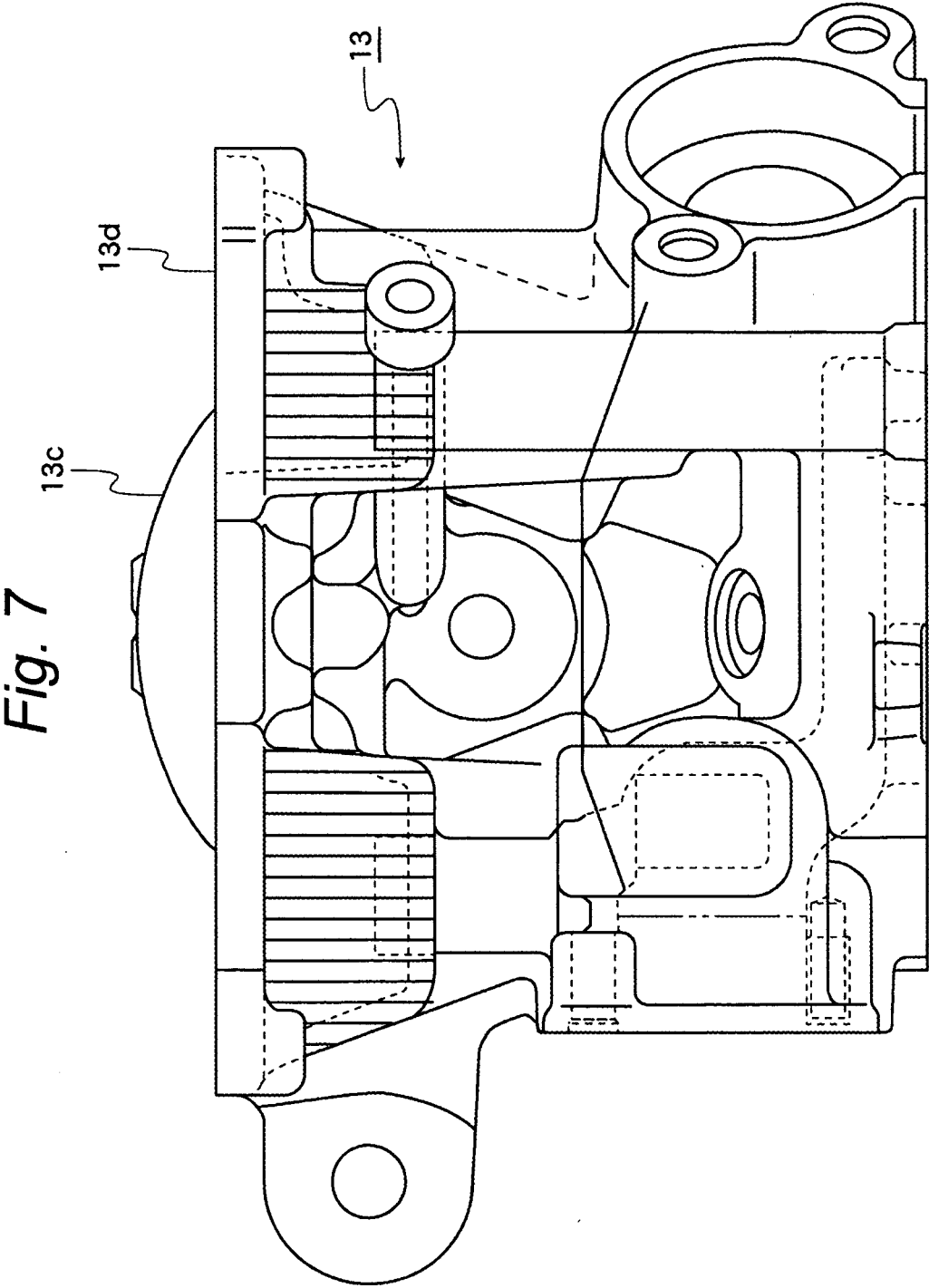


Fig. 6





INTERNATIONAL SEARCH REPORT

International application No.

PCT/JP2004/009899

A. CLASSIFICATION OF SUBJECT MATTER

Int.Cl⁷ F02F1/24

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

Int.Cl⁷ F02F1/00-7/00

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Jitsuyo Shinan Koho	1922-1996	Toroku Jitsuyo Shinan Koho	1994-2004
Kokai Jitsuyo Shinan Koho	1971-2004	Jitsuyo Shinan Toroku Koho	1996-2004

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
Y A	JP 9-195851 A (Honda Motor Co., Ltd.), 29 July, 1997 (29.07.97), Full text; Figs. 1 to 8 (Family: none)	1-3, 6, 8 4, 5, 7
Y A	JP 2002-48000 A (Honda Motor Co., Ltd.), 15 February, 2002 (15.02.02), Full text; Figs. 1 to 8 & CN 1337524 A	1-3, 6, 8 4, 5, 7

☐ Further documents are listed in the continuation of Box C.☐ See patent family annex.

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Date of the actual completion of the international search
14 September, 2004 (14.09.04)Date of mailing of the international search report
28 September, 2004 (28.09.04)Name and mailing address of the ISA/
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