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## (54) SCROLL COMPRESSOR

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## ABSTRACT

The scroll compressor includes scroll members that include a compression chamber to compress working fluid, a housing that houses the scroll members, a second driving-side shaft portion that discharges the compressed working fluid from the compression chamber and is rotated around an axis with respect to the housing, a bearing that rotatably supports the housing with respect to the second driving-side shaft portion, and a seal member between the second driving-side shaft portion and the housing. A lubricant is provided between the bearing and the seal member.

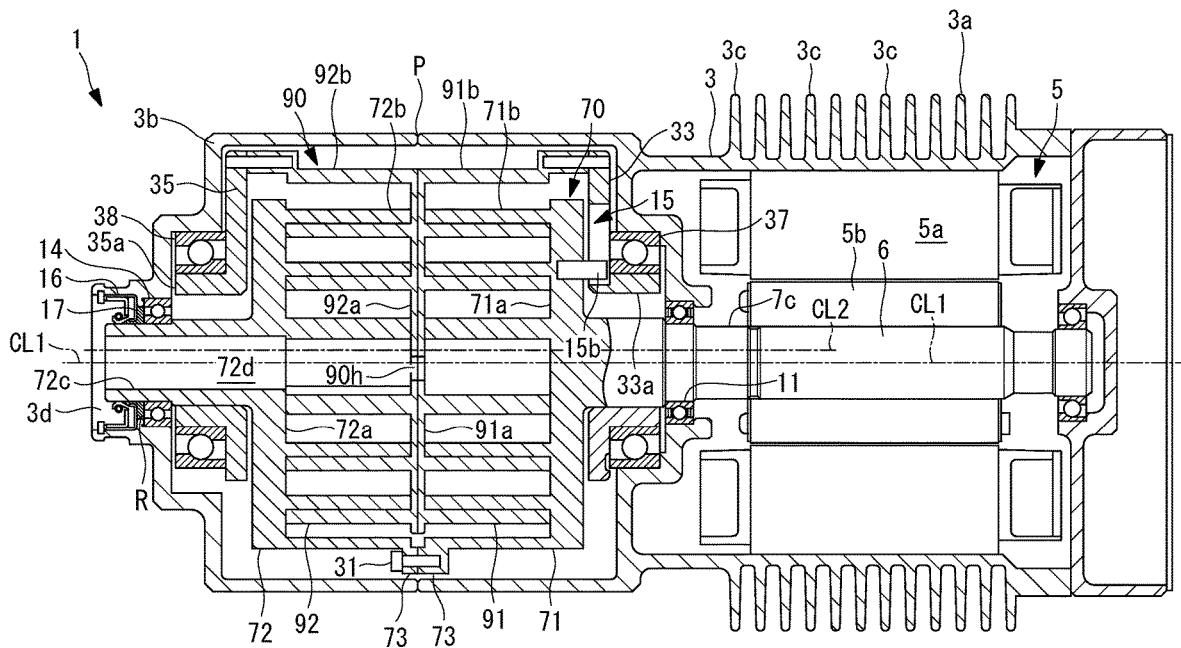


FIG. 1

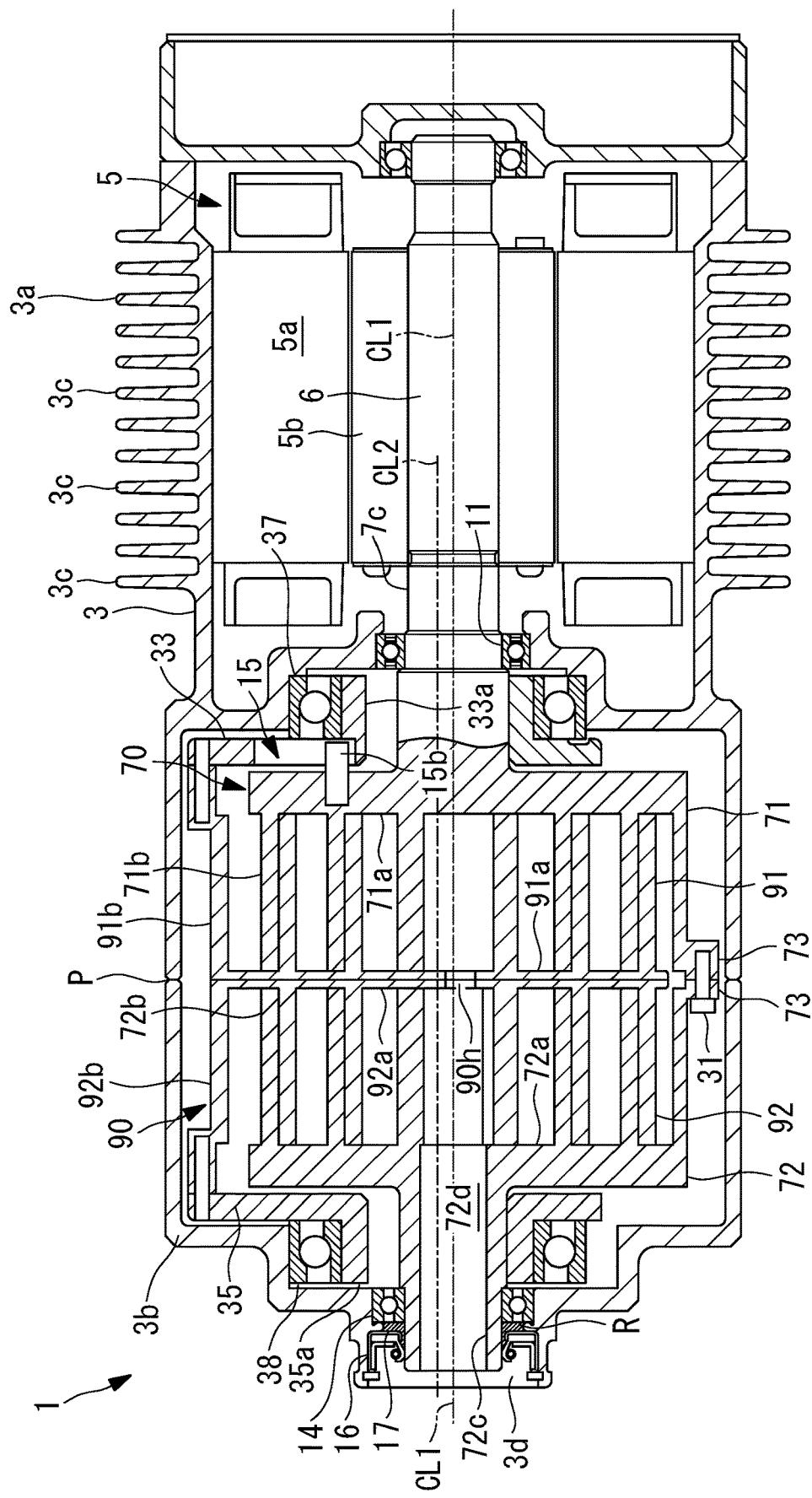


FIG. 2

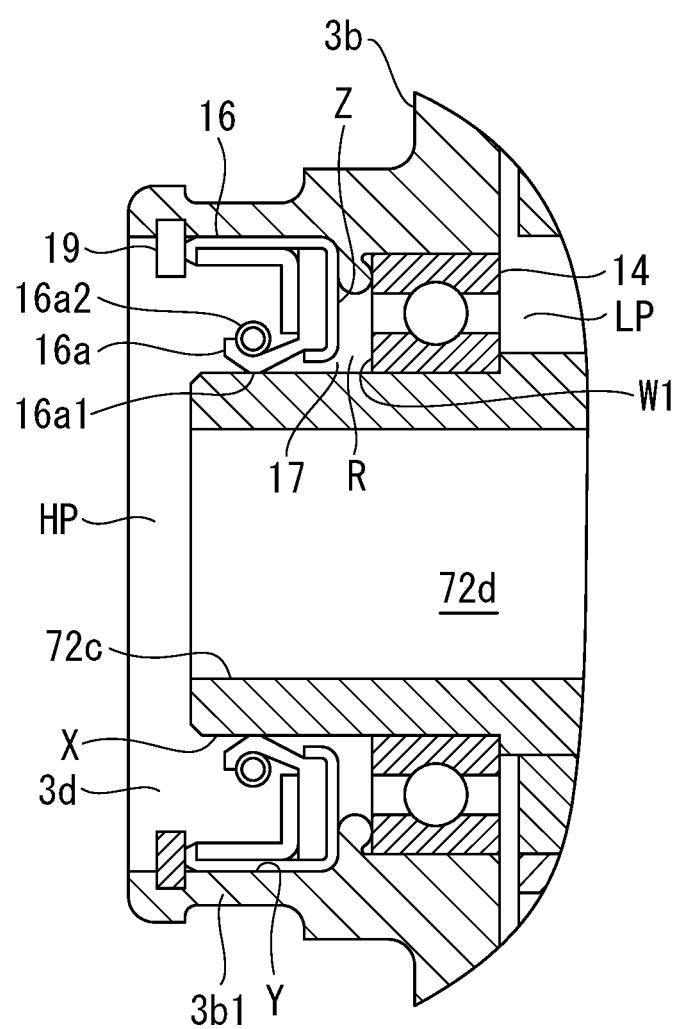


FIG. 3

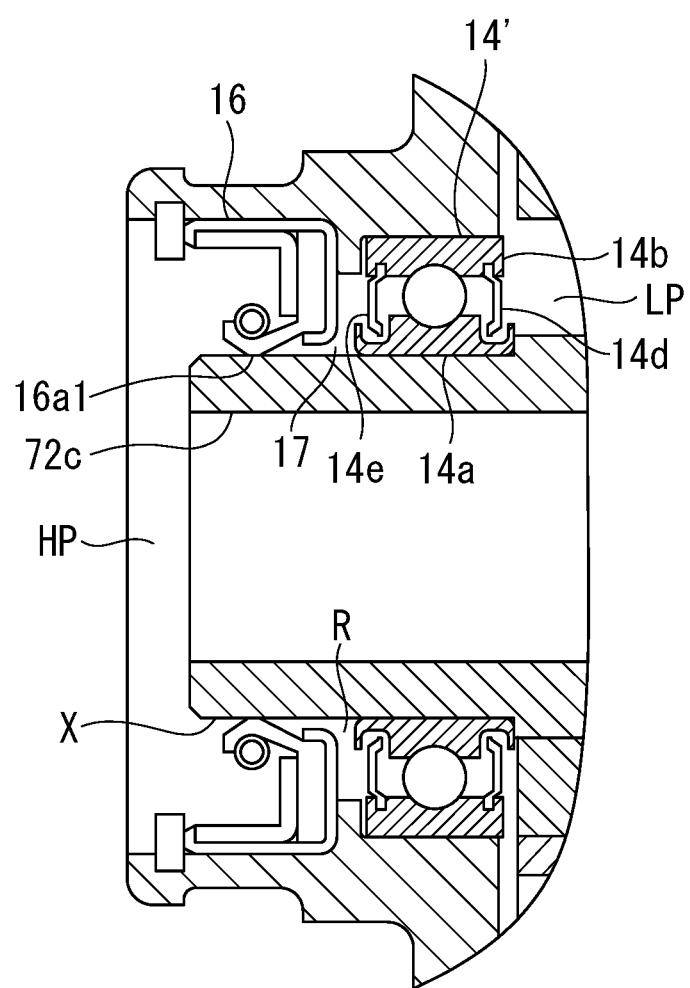


FIG. 4

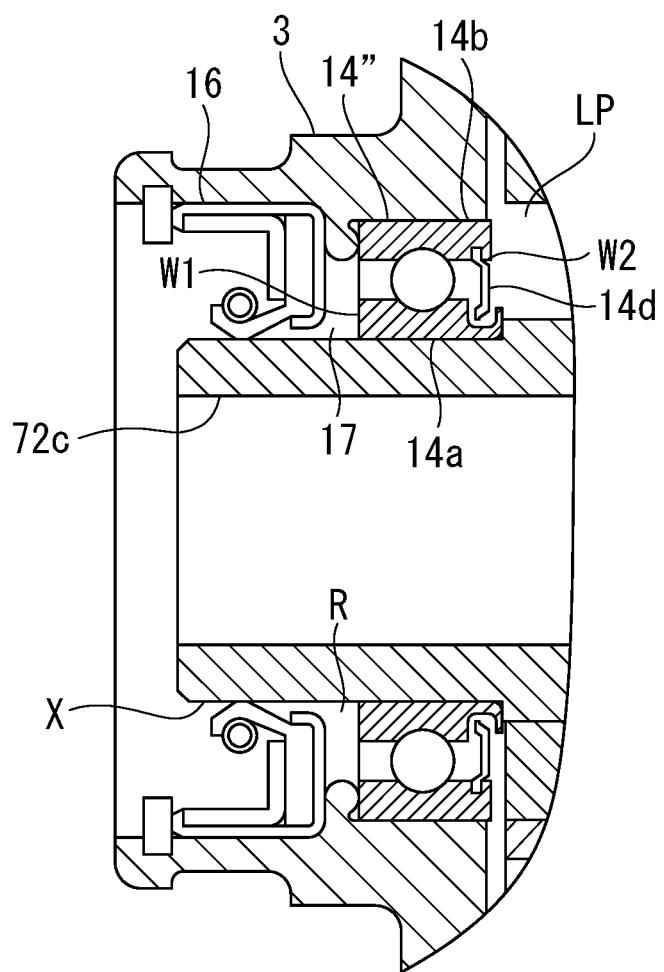
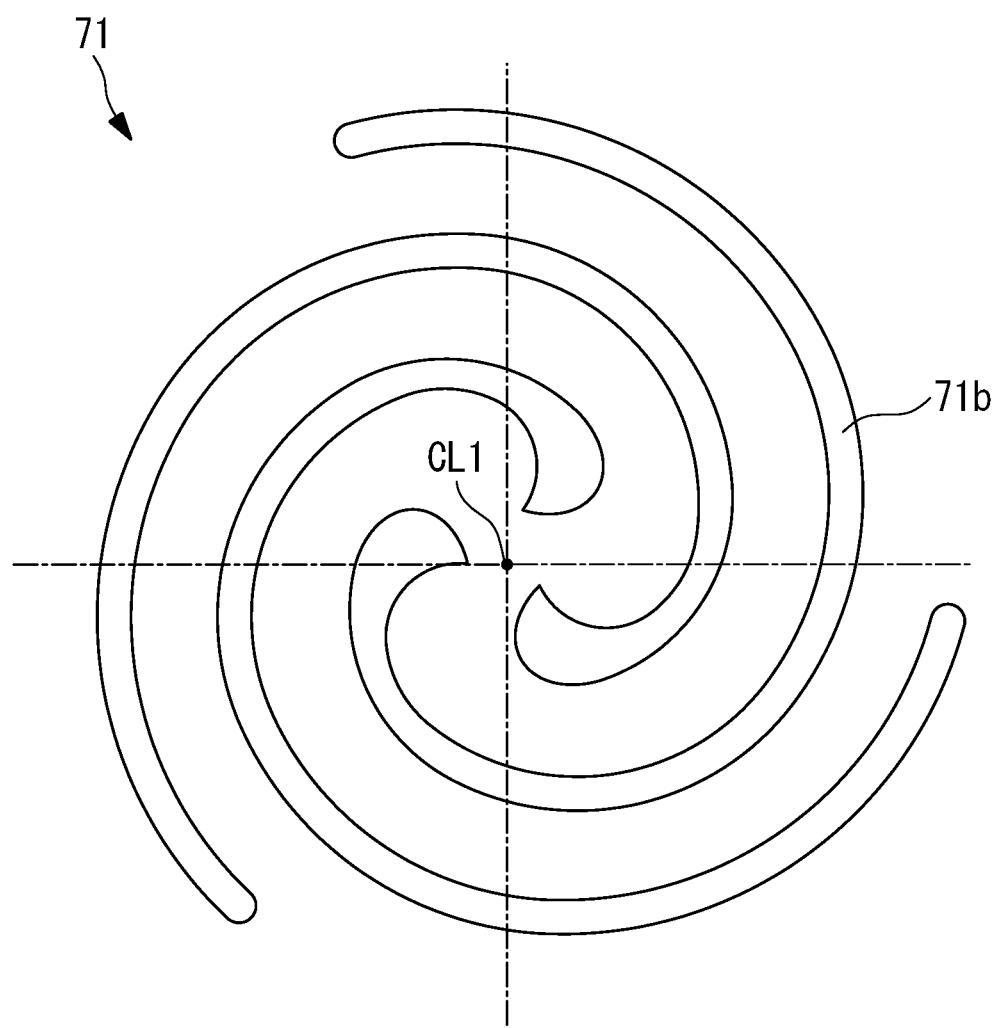


FIG. 5



## SCROLL COMPRESSOR

### TECHNICAL FIELD

**[0001]** The present disclosure relates to a scroll compressor suitably used for, for example, a double rotating scroll compressor.

### BACKGROUND ART

**[0002]** A double rotating scroll compressor has been well-known (refer to PTL 1). Such a double rotating scroll compressor includes a seal plate on a front end surface of a discharge cylinder. The seal plate seals high-pressure side occupied by compressed working fluid and low-pressure side that is an internal space of a housing.

### CITATION LIST

#### Patent Literature

**[0003]** [PTL 1] Japanese Unexamined Patent Application, Publication No. 2015-1175

### SUMMARY OF INVENTION

#### Technical Problem

**[0004]** In PTL 1, a seal plate **48** sandwiched between a front end surface of a discharge cylinder **46** and a boss portion **50** is compressed in an axis direction to achieve sealing (reference numerals are those described in PTL 1). In a case where force pressing the seal plate in the axis direction is not sufficient, however, sealability may be deteriorated.

**[0005]** In contrast, it is conceivable to have a configuration to seal an outer peripheral surface of the discharge cylinder that is rotated around the axis. When a seal member is disposed on an outer periphery of the discharge cylinder, however, an axial dimension of the discharge cylinder is increased and pressure loss is increased.

**[0006]** Further, in a case of an oilless compressor in which no misty oil is contained in the fluid to be compressed, it is necessary to supply a lubricant to lubricate the sealing member.

**[0007]** The present disclosure is made in consideration of such circumstances, and an object of the present disclosure is to provide a scroll compressor that makes it possible to reduce the axial dimension of the discharge cylinder rotated around the axis while supplying a lubricant when the outer peripheral surface of the discharge cylinder is sealed.

#### Solution to Problem

**[0008]** To solve the above-described issues, a scroll compressor according to the present disclosure adopts the following solutions. The scroll compressor according to the present disclosure includes paired scroll members that include a compression chamber to compress working fluid, a housing that houses the paired scroll members, a discharge cylinder that discharges the compressed working fluid from the compression chamber and is rotated around an axis with respect to the housing, a bearing that rotatably supports the discharge cylinder with respect to the housing, and a seal member that is provided on outlet side of the discharge cylinder relative to the bearing and seals an outer peripheral

surface of the discharge cylinder. A lubricant is enclosed in a space formed between the bearing and the seal member.

**[0009]** The space is formed between the seal member and the bearing that rotatably supports the discharge cylinder, and the lubricant is enclosed in the space. This makes it possible to configure the space in which the lubricant is enclosed, without waste. For example, as compared with a configuration in which two seal members are disposed side by side in the axis direction and the lubricant is enclosed in a space between the seal members, it is sufficient to provide one seal member, and it is possible to reduce the number of components and to reduce the axial dimension of the discharge cylinder. Accordingly, the outer peripheral surface of the discharge cylinder can be sealed in compact, and pressure loss of the working fluid can be reduced.

**[0010]** In particular, even in an oilless compressor in which the working fluid contains no lubricant oil, lubrication of the seal member is achieved by the lubricant enclosed between the seal member and the bearing, which allows for reduction of wear.

**[0011]** Even when the oilless compressor is used in a case where clean working fluid containing no oil is desired in a demander of the compressed working fluid, the scroll compressor according to the present disclosure can reduce wear of the seal member.

**[0012]** Further, in the scroll compressor according to the present disclosure, the bearing may be a rolling bearing including seal plates on respective side surfaces, and the seal plates may be fixed to an outer ring of the bearing and may be not in contact with an inner ring of the bearing.

**[0013]** In the case where the rolling bearing including the seal plates on respective end surface sides is used for the bearing, such a non-contact system, in which the seal plates do not come into contact with the inner ring, makes it possible to prevent energy loss due to generation of heat and the like due to the contact, and to prevent deterioration of system efficiency of the scroll compressor.

**[0014]** Further, in the scroll compressor according to the present disclosure, the bearing may be a rolling bearing including a seal plate on a side surface on side opposite to the seal member side, and may not include a seal plate on a side surface on the seal member side.

**[0015]** The side surface of the bearing on the seal member side faces the space in which high-pressure working fluid is sufficiently sealed by the seal member and in which the lubricant is enclosed. This allows for omission of the seal plate on the seal member side. On the other hand, a side surface of the bearing on the side opposite to the seal member faces an inside of the housing. Therefore, the seal plate is disposed to prevent leakage of the lubricant.

**[0016]** Further, in the scroll compressor according to the present disclosure, the seal member may be an oil seal.

**[0017]** The lubricant is provided between the oil seal and the bearing, which makes it possible to reduce wear of the oil seal.

**[0018]** Furthermore, the scroll compressor according to the present disclosure may further include a driving shaft that is rotationally driven by a driving unit, and the scroll compressor may be configured as a double rotating scroll compressor that includes a driving-side scroll member and a driven-side scroll member as the paired scroll members. The driving-side scroll member is coupled to the driving shaft and performs rotational movement, and the driven-side

scroll member receives power transmitted from the driving-side scroll member to perform rotational movement.

[0019] In the double rotating scroll compressor not including a static scroll member, possibility of wear of the seal member is remarkable. The wear can be reduced by providing the lubricant between the seal member and the bearing.

#### Advantageous Effects of Invention

[0020] It is possible to reduce the axial dimension of the discharge cylinder rotated around the axis while supplying the lubricant when the outer peripheral surface of the discharge cylinder is sealed.

#### BRIEF DESCRIPTION OF DRAWINGS

[0021] FIG. 1 is a vertical cross-sectional view illustrating a double rotating scroll compressor according to an embodiment of the present disclosure.

[0022] FIG. 2 is a vertical cross-sectional view illustrating a main part of FIG. 1 in an enlarged manner.

[0023] FIG. 3 is a vertical cross-sectional view illustrating a modification 1.

[0024] FIG. 4 is a vertical cross-sectional view illustrating a modification 2.

[0025] FIG. 5 is a plan view illustrating a driving-side scroll member.

#### DESCRIPTION OF EMBODIMENTS

[0026] An embodiment of the present disclosure is described below with reference to FIG. 1 and FIG. 2.

[0027] FIG. 1 illustrates a double rotating scroll compressor (scroll compressor) 1. The double rotating scroll compressor 1 can be used as, for example, a supercharger that compresses combustion air (fluid) to be supplied to an internal combustion engine such as a vehicle engine.

[0028] The double rotating scroll compressor 1 includes a housing 3, a motor (driving unit) 5 accommodated on one end side in the housing 3, and a driving-side scroll member 70 and a driven-side scroll member 90 that are accommodated on the other end side in the housing 3.

[0029] The housing 3 has a substantially cylindrical shape, and includes a motor accommodation portion (first housing) 3a that accommodates the motor 5, and a scroll accommodation portion (second housing) 3b that accommodates the scroll members 70 and 90.

[0030] A cooling fin 3c to cool the motor 5 is provided on an outer periphery of the motor accommodation portion 3a. A discharge opening 3d from which compressed air (working fluid) is discharged is provided at an end part of the scroll accommodation portion 3b. Note that, although not illustrated in FIG. 1, the housing 3 includes an air suction opening from which air (working fluid) is sucked in.

[0031] The scroll accommodation portion 3b of the housing 3 is divided at a division surface P that is located at a substantially center in an axis direction of the scroll members 70 and 90. The housing 3 includes a flange portion (not illustrated) that protrudes outward at a predetermined position in a circumferential direction. A bolt or the like as a fastening means is inserted into and fixed to the flange portion, which results in fastening at the division surface P.

[0032] The motor 5 is driven by being supplied with power from an unillustrated power supply source. Rotation of the motor 5 is controlled by an instruction from an unillustrated control unit. A stator 5a of the motor 5 is fixed

to an inner periphery of the housing 3. A rotor 5b of the motor 5 rotates around a driving-side rotation axis CL1. A driving shaft 6 that extends on the driving-side rotation axis CL1 is connected to the rotor 5b. The driving shaft 6 is connected to a first driving-side shaft portion 7c of the driving-side scroll member 70.

[0033] The driving-side scroll member 70 includes a first driving-side scroll portion 71 on the motor 5 side, and a second driving-side scroll portion 72 on the discharge opening 3d side.

[0034] The first driving-side scroll portion 71 includes a first driving-side end plate 71a and first driving-side walls 71b.

[0035] The first driving-side end plate 71a is connected to the first driving-side shaft portion 7c connected to the driving shaft 6, and extends in a direction orthogonal to the driving-side rotation axis CL1. The first driving-side shaft portion 7c is provided so as to be rotatable with respect to the housing 3 through a first driving-side bearing 11 that is a ball bearing.

[0036] The first driving-side end plate 71a has a substantially disc shape in a planar view. The first driving-side wall 71b formed in a spiral shape is provided on the first driving-side end plate 71a. The three lines of first driving-side walls 71b are disposed at an equal interval around the driving-side rotation axis CL1 (see FIG. 5).

[0037] As illustrated in FIG. 1, the second driving-side scroll portion 72 includes a second driving-side end plate 72a and second driving-side walls 72b. The second driving-side walls 72b are each formed in a spiral shape as with the above-described first driving-side walls 71b.

[0038] A second driving-side shaft portion (discharge cylinder) 72c that extends in the driving-side rotation axis CL1 and has a cylindrical shape is connected to the second driving-side end plate 72a. The second driving-side shaft portion 72c is provided so as to be rotatable with respect to the housing 3 through a second driving-side bearing 14 that is a ball bearing. The second driving-side end plate 72a includes a discharge port 72d extending along the driving-side rotation axis CL1.

[0039] A seal member 16 is provided on downstream side of the second driving-side bearing 14 in a discharge direction of the compressed air, namely, on front end side that is a free end of the second driving-side shaft portion 72c, between the second driving-side shaft portion 72c and the housing 3. The seal member 16 and the second driving-side bearing 14 are disposed at a predetermined interval in the driving-side rotation axis CL1 direction, and a space R is provided therebetween. A lubricant 17 that is grease as, for example, semi-solid lubricant is enclosed in the space R.

[0040] The first driving-side scroll portion 71 and the second driving-side scroll portion 72 are fixed while front ends (free ends) of the walls 71b and 72b corresponding to each other face each other. The first driving-side scroll portion 71 and the second driving-side scroll portion 72 are fixed by bolts (wall fixing parts) 31 that are fastened to respective flange portions 73 provided at a plurality of positions in the circumferential direction. The flange portions 73 are provided so as to protrude outward in a radial direction.

[0041] The driven-side scroll member 90 includes a first driven-side scroll portion 91 and a second driven-side scroll portion 92. Driven-side end plates 91a and 92a are located at a substantially center of the driven-side scroll member 90

in the axis direction (horizontal direction in figure). The driven-side end plates **91a** and **92a** are fixed while rear surfaces (other side surfaces) of the respective driven-side end plates **91a** and **92a** are superimposed and in contact with each other. Although not illustrated, the fixing is performed by a bolt, a pin, etc. The through hole **90h** is provided at a center of each of the driven-side end plates **91a** and **92a**, and causes the compressed air to flow toward the discharge port **72d**.

[0042] The first driven-side walls **91b** are provided on one side surface of the first driven-side end plate **91a**, and the second driven-side walls **92b** are provided on one side surface of the second driven-side end plate **92a**. The first driven-side walls **91b** provided on the motor **5** side from the first driven-side end plate **91a** engage with the first driving-side walls **71b** of the first driving-side scroll portion **71**. The second driven-side walls **92b** provided on the discharge opening **3d** side from the second driven-side end plate **92a** engage with the second driving-side walls **72b** of the second driving-side scroll portion **72**.

[0043] The support members **33** and **35** described later are fixed to the outer peripheries of the first driven-side walls **91b**. The second driven-side walls **92b** also have the similar configuration.

[0044] The first support member **33** and the second support member **35** are provided at the respective ends of the driven-side scroll member **90** in the axis direction (horizontal direction in figure). The first support member **33** is disposed on the motor **5** side, and the second support member **35** is disposed on the discharge opening **3d** side. The first support member **33** is fixed to the front ends (free ends) of the first driven-side walls **91b**, and the second support member **35** is fixed to the front ends (free ends) of the second driven-side walls **92b**. The shaft portion **33a** is provided on the center axis side of the first support member **33**, and the shaft portion **33a** is fixed to the housing **3** through the first support member bearing **37**. The shaft portion **35a** is provided on the center axis side of the second support member **35**, and the shaft portion **35a** is fixed to the housing **3** through the second support member bearing **38**. As a result, the driven-side scroll member **90** rotates around the driven-side rotation axis **CL2** through the support members **33** and **35**.

[0045] The pin-ring mechanism (synchronous driving mechanism) **15** is provided between the first support member **33** and the first driving-side end plate **71a**. More specifically, a circular hole is provided in the first driving-side end plate **71a**, and the pin member **15b** is provided on the first support member **33**. The pin-ring mechanism **15** transmits the driving force from the driving-side scroll member **70** to the driven-side scroll member **90**, and causes the scroll members **70** and **90** to perform rotational movement in the same direction at the same angular velocity.

[0046] As illustrated in FIG. 2, the scroll accommodation portion **3b** includes a second driving-side shaft portion accommodation portion **3b1** that accommodates the second driving-side shaft portion **72c**, and the second driving-side bearing **14** is provided between an outer peripheral surface **X** of the second driving-side shaft portion **72c** and an inner peripheral surface **Y** of the second driving-side shaft portion accommodation portion **3b1**.

[0047] The seal member **16** is an oil seal, and a position of the seal member **16** in the axis direction is regulated by a stopper ring **19** that is fitted in the inner peripheral surface

**Y** of the second driving-side shaft portion accommodation portion **3b1** as illustrated in FIG. 2. The seal member **16** includes a seal lip portion **16a** made of a resin. The seal lip portion **16a** includes a lip front end part **16a1** that protrudes to the inner peripheral side and comes into contact with the outer peripheral surface **X** of the second driving-side shaft portion **72c**. An annular spring **16a2** is provided on rear-surface side (outer peripheral side) of the seal lip portion **16a**. The lip front end part **16a1** is pressed against the entire circumference of the outer peripheral surface **X** of the second driving-side shaft portion **72c** by elastic force of the spring **16a2**.

[0048] The lubricant **17** is enclosed in the space **R** surrounded by a surface of the seal member **16** and a surface of the second driving-side bearing **14** facing each other, namely, a side surface **Z** of the seal member **16** and a surface **W1** of the second driving-side bearing **14**, the outer peripheral surface **X** of the second driving-side shaft portion **72c**, and the inner peripheral surface **Y** of the second driving-side shaft portion accommodation portion **3b1**.

[0049] The double rotating scroll compressor **1** including the above-described configuration operates in the following manner.

[0050] When the driving shaft **6** rotates around the driving-side rotation axis **CL1** by the motor **5**, the first driving-side shaft portion **7c** connected to the driving shaft **6** also rotates, and the driving-side scroll member **70** accordingly rotates around the driving-side rotation axis **CL1**. When the driving-side scroll member **70** rotates, the driving force is transmitted from the support members **33** and **35** to the driven-side scroll member **90** through the pin-ring mechanisms **15**, and the driven-side scroll member **90** rotates around the driven-side rotation axis **CL2**. At this time, when the pin member **15b** of the pin-ring mechanism **15** moves while being in contact with the inner peripheral surface of the circular hole, the both scroll members **70** and **90** perform rotational movement in the same direction at the same angular velocity.

[0051] When the scroll members **70** and **90** perform rotational movement, the air sucked through the air suction opening of the housing **3** is sucked in from outer peripheral side of each of the scroll members **70** and **90**, and is taken into the compression chambers formed by the scroll members **70** and **90**. Further, compression is separately performed in the compression chambers formed by the first driving-side walls **71b** and the first driven-side walls **91b** and in the compression chambers formed by the second driving-side walls **72b** and the second driven-side walls **92b**. A volume of each of the compression chambers is reduced as each of the compression chambers moves toward the center, which compresses the air. The air compressed by the first driving-side walls **71b** and the first driven-side walls **91b** passes through the through holes **90h** provided in the driven-side end plates **91a** and **92a**, and is joined with the air compressed by the second driving-side walls **72b** and the second driven-side walls **92b**. The resultant air passes through the discharge port **72d** and is discharged to outside from the discharge opening **3d** of the housing **3**. The discharged compressed air is guided to an unillustrated internal combustion engine, and is used as combustion air.

[0052] A high-pressure space **HP** occupied by the compressed air that has been discharged from the discharge port **72d** but before being discharged to the outside from the discharge opening **3d** and a low-pressure space **LP** occupied

by sucked air that is sucked from the suction opening of the housing 3 and is taken in from the outer peripheral side of the both scroll members 70 and 90 are partitioned by the seal member 16, the lubricant 17, and the second driving-side bearing 14.

[0053] The lip front end part 16a1 that is a front end of the seal lip portion 16a of the seal member 16 is pressed against the outer peripheral surface X of the second driving-side shaft portion 72c by the spring 16a2 provided on the seal lip portion 16a, and an oil film is formed by the lubricant 17 between the lip front end part 16a1 and the outer peripheral surface X of the second driving-side shaft portion 72c.

[0054] The present embodiment achieves the following action effects.

[0055] The space R is formed between the seal member 16 and the second driving-side bearing 14 that rotatably supports the second driving-side shaft portion 72c, and the lubricant 17 is enclosed in the space R. Therefore, it is possible to configure the space in which the lubricant 17 is enclosed, without waste. For example, as compared with a configuration in which two seal members are disposed side by side in the axis direction and the lubricant is enclosed in a space between the seal members, it is possible to reduce the number of components and to reduce an axial dimension of the second driving-side shaft portion 72c. Accordingly, the outer peripheral surface X of the second driving-side shaft portion 72c can be sealed in compact, which makes it possible to reduce pressure loss of the compressed air.

[0056] In particular, even in an oilless compressor in which the air does not contain the lubricant oil, lubrication of the seal member 16 is achieved by the lubricant 17 enclosed between the seal member 16 and the second driving-side bearing 14, which allows for reduction of wear.

[0057] Even when the oilless compressor is used in a case where clean compressed air containing no oil is desired in a demander of the compressed air, the scroll compressor 1 according to the present embodiment can reduce wear of the seal member.

[0058] The present embodiment may be modified in the following manner.

#### Modification 1

[0059] As illustrated in FIG. 3, a sealed ball bearing may be used as the second driving-side bearing 14. A second driving-side bearing 14' includes seal plates 14d and 14e on respective side surfaces. The seal plates 14d and 14e prevent leakage of the lubricant oil inside the second driving-side bearing 14'. Each of the seal plates 14d and 14e is formed in an annular plate shape, and a seal plate in which a steel plate is coated with rubber is suitably used for each of the seal plates 14d and 14e. An outer periphery of each of the seal plates 14d and 14e is fixed to an outer ring 14b of the second driving-side bearing 14', and an inner periphery of each of the seal plates 14d and 14e is separated from an inner ring 14a so as not to come into contact with the inner ring 14a.

[0060] As described above, a non-contact configuration in which the seal plates 14d and 14e do not come into contact with the inner ring 14a makes it possible to prevent energy loss due to generation of heat and the like due to the contact, thereby preventing deterioration of system efficiency of the scroll compressor 1.

#### Modification 2

[0061] As illustrated in FIG. 4, a second driving-side bearing 14" may include only the seal plate 14d on side W2 opposite to the seal member 16 without the seal plate 14e on the seal member side W1 illustrated in FIG. 3.

[0062] A side surface of the bearing 14" on the seal member side W1 faces the space R in which the high-pressure air of the high-pressure space HP is sufficiently sealed by the seal member 16 and in which the lubricant 17 is enclosed. This allows for omission of the seal plate on the seal member side W1. On the other hand, a side surface of the bearing 14" on the side W2 opposite to the seal member 16 faces the inside of the housing 3. Therefore, the seal plate 14d is disposed to prevent leakage of the lubricant.

[0063] In the present embodiment, the oil seal is used as the seal member 16; however, the oil seal may be replaced with a self-seal packing such as a gasket, a mechanical seal, a piston ring, and an O-ring. The lubricant 17 can reduce wear at a contact portion between the seal member 16 that is a self-seal packing such as a gasket, a mechanical seal, a piston ring, and an O-ring and the outer peripheral surface X of the second driving-side shaft portion 72c.

[0064] Further, grease that is semi-solid lubricant has been used as the lubricant 17. Alternatively, lubricant oil that is fluid lubricant and is represented by mineral oil may be used.

[0065] The above-described scroll compressor 1 may be an "oilless" or "oil-free" compressor in which the compressed air that has been compressed in the compression chamber formed by the scroll members 7 and 9 contains no lubricant oil, and the compressed air is discharged to the outside from the discharge opening 3d through the discharge port 72d. In the case of the oilless compressor, the seal member 16 may be worn down because no lubricant oil is contained in the housing 3 including the high-pressure space HP and the low-pressure space LP; however, providing the lubricant 17 to the space between the seal member 16 and the second driving-side bearing 14 makes it possible to achieve lubrication of the seal member 16 and to reduce wear of the seal member 16.

[0066] Note that, in the above-described embodiments and modifications, the double rotating scroll compressor is used as the supercharger; however, the present disclosure is not limited thereto. The double rotating scroll compressor is widely used to compress fluid, and for example, can be used as a refrigerant compressor used in air conditioner. In addition, the scroll compressor 1 according to the present disclosure is applicable to an air brake device using air force, as a brake system for a railway vehicle.

#### REFERENCE SIGNS LIST

- [0067] 1 Double rotating scroll compressor (scroll compressor)
- [0068] 3 Housing
- [0069] 3a Motor accommodation portion
- [0070] 3b Scroll accommodation portion
- [0071] 3b1 Second driving-side shaft portion accommodation portion
- [0072] 3c Cooling fin
- [0073] 3d Discharge opening
- [0074] 5 Motor (driving unit)
- [0075] 5a Stator
- [0076] 5b Rotor
- [0077] 6 Driving shaft

[0078]	7c First driving-side shaft portion	[0122]	Z Side surface of seal member
[0079]	11 First driving-side bearing	[0123]	W1 Seal member side of second driving-side bearing
[0080]	14, 14', 14" Second driving-side bearing	[0124]	W2 Side of second driving-side bearing opposite to seal member
[0081]	14a Inner ring	[0125]	HP High-pressure space
[0082]	14b Outer ring	[0126]	LP Low-pressure space
[0083]	14d Seal plate	[0127]	R Space
[0084]	14e Seal plate		
[0085]	15 Pin-ring mechanism (synchronous driving mechanism)		1. A scroll compressor, comprising:
[0086]	15b Pin member		paired scroll members that include a compression chamber to compress working fluid;
[0087]	16 Seal member (oil seal)		a housing that houses the paired scroll members;
[0088]	16a Seal lip portion		a discharge cylinder that discharges the compressed working fluid from the compression chamber and is rotated around an axis with respect to the housing;
[0089]	16a1 Lip front end part		a bearing that rotatably supports the discharge cylinder with respect to the housing; and
[0090]	16a2 Spring		a seal member that is located between the discharge cylinder and the housing, wherein
[0091]	17 Lubricant (grease)		a lubricant is enclosed in a space formed between the bearing and the seal member.
[0092]	31 Bolt (wall fixing part)		2. The scroll compressor according to claim 1, wherein the bearing is a rolling bearing including seal plates on respective side surfaces, and
[0093]	33 First support member		the seal plates are fixed to an outer ring of the bearing and are not in contact with an inner ring of the bearing.
[0094]	33a Shaft portion		3. The scroll compressor according to claim 1, wherein the bearing is a rolling bearing including a seal plate on a side surface on side opposite to the seal member side, and does not include a seal plate on a side surface on the seal member side.
[0095]	35 Second support member		4. The scroll compressor according to claim 1, wherein the seal member is an oil seal.
[0096]	35a Shaft portion		5. The scroll compressor according to claim 1, further comprising a driving shaft that is rotationally driven by a driving unit, wherein
[0097]	37 First support member bearing		the scroll compressor is configured as a double rotating scroll compressor that includes a driving-side scroll member and a driven-side scroll member as the paired scroll members, the driving-side scroll member being coupled to the driving shaft and performing rotational movement, and the driven-side scroll member receiving power transmitted from the driving-side scroll member to perform rotational movement.
[0098]	38 Second support member bearing		
[0099]	70 Driving-side scroll member		
[0100]	71 First driving-side scroll portion		
[0101]	71a First driving-side end plate		
[0102]	71b First driving-side wall		
[0103]	72 Second driving-side scroll portion		
[0104]	72a Second driving-side end plate		
[0105]	72b Second driving-side wall		
[0106]	72c Second driving-side shaft portion		
[0107]	72d Discharge port		
[0108]	73 Flange portion		
[0109]	90 Driven-side scroll member		
[0110]	90h Through hole		
[0111]	91 First driven-side scroll portion		
[0112]	91a First driven-side end plate		
[0113]	91b First driven-side wall		
[0114]	92 Second driven-side scroll portion		
[0115]	92a Second driven-side end plate		
[0116]	92b Second driven-side wall		
[0117]	CL1 Driving-side rotation axis		
[0118]	CL2 Driven-side rotation axis		
[0119]	P Division surface		
[0120]	X Outer peripheral surface of second driving-side shaft portion		
[0121]	Y Inner peripheral surface of second driving-side shaft portion accommodation portion		

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