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- (54) Benævnelse: **Centrifugalseparator, slidbestandigt element og sæt af slidbestandige elementer til en centrifugalseparator**
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A centrifugal separator comprises a bowl with a conical part (17) with a narrow discharge end (15) comprising a radial surface (19); an end member (21) opposite the radial surface; a number of distance members extending between the radial surface (19) and the end member (21) and providing outlet openings (10) between adjacent distance members (27); and wear resistance members covering surfaces at the outlet openings, wherein the wear resistance members comprise bushing members (29) with mantle portions (31) surrounding respective distance members (27).

Fortsættes ...

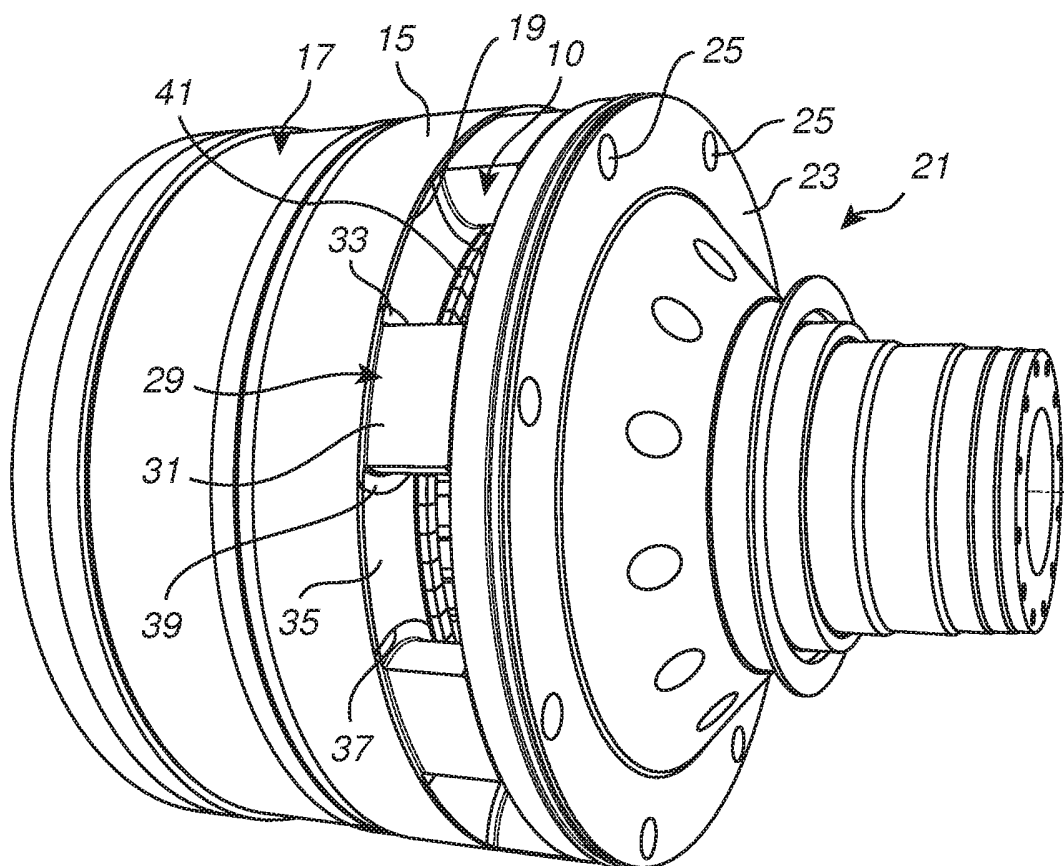


Fig. 2

The present invention relates to a centrifugal separator comprising a bowl rotating in use in a direction of rotation around an axis of rotation extending in a longitudinal direction of the bowl, said bowl comprising a conical part with a narrow discharge end comprising a radial surface; an end member opposite the radial surface; a number of distance members extending between the radial surface and the end member and providing outlet openings between adjacent distance members, said distance members having an axial extend in an axial direction of the axis of rotation and a circumferential extend perpendicular to the axial extend; and wear resistance members covering surfaces at the outlet openings.

The invention further relates to a wear resistance member for such centrifugal separator and to a set of wear resistance members for a centrifugal separator.

It is well known to provide wear resistance members at the solid phase or heavy phase outlet of a centrifugal separator since the solid phase separated from a feed inside the bowl of the centrifugal separator is abrasive in some applications like e.g. drilling mud.

US-A-5 244 584 discloses a centrifugal separator of the art mentioned wherein L-shaped wear resistance members are provided to protect the surfaces of the distance members directly exposed to the out-flowing solid phase. The leg of the L-shaped member covering the leading surface of the distance member extends beyond the external surface of the bowl. The L-shaped member is fastened by a bolt inserted through the distance member and into the L-shaped member from the outside, and the L-shaped member is dimensioned in a way so that it can be inserted and fitted without the bowl having to be dismantled. The L-shaped members are of a complicated construction involving many parts to be assembled.

The bowl is usually accommodated in a casing with compartments receiving the material discharged from the outlets of the bowl e.g. the solid phase. The solid phase may build-up in the receiving compartment until it reaches the outside of the bowl at the outlets causing abrasion of the bowl. The leg of the L-shaped member extending beyond the

external surface of the bowl may act as a scraper to reduce this problem. The L-shaped members are build from several pieces and thus have a complicated construction.

US-B-7 374 529 discloses another centrifugal separator of the
5 above mentioned kind wherein U-shaped wear resistance members are inserted in the outlet openings from the outside and fastened from the outside by means of bolts inserted through external flanges of the U-shaped member and fastened in the material of the bowl. The bolts thus have to counteract the centrifugal force acting on the U-shaped member.
10 The U-shaped members extend beyond the external surface of the bowl and between the U-shaped members spacers are fastened to the external side of the distance members by means of bolts counteracting the centrifugal force acting on the spacers. Since the material used for wear resistance members are usually brittle using bolts introduced through
15 the wear resistance member from the outside may cause a strength problem.

The object of the present invention is to avoid or reduce the problems outlined above.

This is obtained in that the wear resistance members comprise a
20 bushing member with a mantle portion surrounding a respective distance member at least around a sufficient portion of the circumferential extend of said distance member to prevent removal of the bushing member in a direction perpendicular to the axial direction. Hereby is i.a. obtained that the need for bolts penetrating the wear resistance mem-
25 bers from the outside to counteract the centrifugal force is avoided, because the bushing members are securely carried by the distance members. Providing wear members in the form of bushings entails that these wear members must be fitted on the distance members before the end member of the centrifugal separator is assembled with the conical part
30 of the bowl, and correspondingly dismantling is necessary in order to renew the wear members. However, using a high quality wear resistance material renewal is needed only at such long intervals, that this drawback is inferior to the benefits of the invention.

In one embodiment the mantle portion of the bushing member

is tubular and circumferentially surrounds the distance member completely. Hereby is obtained an effective protection the outside of the distance members against abrasion.

5 In another embodiment the mantle portion of the bushing member has a C-shaped cross-section perpendicular to the axial direction. Further the wear resistance members preferably comprise a plurality of plate members each covering a portion of the radial surface. Hereby is obtained a simple construction facilitating use of high quality wear resistance materials, such as tungsten carbide.

10 In one embodiment the bushing member has at one end a flange abutting the radial surface and covering a portion of said radial surface. Preferably a plate member is fitted on the radial surface between adjacent bushing members, edges of the flanges of the bushing members overlapping edges of the plate members. Hereby the plate
15 members are retained against the radial surface by the bushing members. Preferably the plate members have a waist portion, the flanges being configured to engage the waist portion and secure the plate members due to the overlap and the waist portion. Hereby the plate members are retained also in both radial directions.

20 In an embodiment the plate members respectively comprise an opening for a distance member to extend through, and an edge of a plate member overlaps an adjacent edge of an adjacent plate member. Preferably the plate member comprises a hole for a fastening member, and the opening for a distance member is adapted to accommodate an
25 end of a bushing member. Due to the distance member and further the bushing member passing through and into, respectively, the opening in the plate member, the latter is retained against the centrifugal force by the distance member together with the bushing member. Preferably the plate members are in this embodiment mounted e.g. by means of glue
30 on a flat steel ring with corresponding holes and a openings, whereby the steel ring with the plate members is fastened by means of e.g. bolts inserted through the holes and tightened against rims of the holes in the steel ring with out being tightened against the rims of the holes of the plate members the latter holes having a larger diameter than the former

holes. The fastening member, e.g. a bolt, inserted through the hole in the plate member will only actively retain the plate member in case the glue fails and then mainly retain the plate member in an axial direction, in which direction the force acting on the plate member is small relative to the centrifugal force.

In an embodiment a concave edge of the wear resistance members covering a portion of the radial surface comprise an extension rising from a plane of the wear resistance member, whereby the extension is arranged to extend around an internal edge of the conical part adjacent the radial surface to protect said internal edge. Hereby protection of an internal edge of the conical part adjacent the radial surface is obtained.

Preferably the wear resistance members comprise tungsten carbide.

Preferably the distance members, and accordingly the mantle portions of the bushing members, have a non-circular cross-section perpendicular to the axial direction preventing the bushing members from rotating around the distance members.

Preferably the mantle portion fits on the distance member with a loose fit and a filling material is provided filling the gaps between the mantle portion and the distance member. By the loose fit is obtained that tensions are avoided that might cause fraction of the bushing members and by the filling material, such as glue, is obtained an even distribution of the forces acting between the mantle portion and the distance member.

The object of the present invention is further obtained by a wear resistance member for a centrifugal separator according to the invention, said wear resistance member being shaped as a bushing member having a mantle portion for surrounding a respective distance member at least around a sufficient portion of a circumferential extend of said distance member to prevent removal of the bushing member in a direction perpendicular to an axial direction, and by a set of wear resistance members for a centrifugal separator according to the invention, the set comprising bushing members having mantle portions for surrounding a respective distance member at least around a sufficient portion of a

circumferential extend of said distance member to prevent removal of the bushing member in a direction perpendicular to an axial direction, and plate members for covering a portion of the radial surface between adjacent bushing members.

5 The invention will in the following be described in further detail by way of example with reference to the attached schematic drawings, where

Fig. 1 shows a centrifugal separator omitting the novel features of the invention,

10 Fig. 2 is a partial oblique view of the narrow end of the conical part of the bowl and the end member fitted with wear resistance members according to a first embodiment of the present invention,

Fig. 3 is a cross section of a bushing member fitted on a distance member,

15 Fig. 4 is an oblique view of a second embodiment of a bushing member,

Fig. 5 is an oblique view of a second embodiment of a plate member,

Fig. 6 is a plan view of the plate member of Fig. 5,

20 Fig. 7 shows a section along line VII-VII in Fig. 6,

Fig. 8 shows plate members of the second embodiment assembled into a ring, and

Fig. 9 shows an oblique view of a third embodiment.

25 A rotating body 1 of a centrifugal separator or decanter centrifuge schematically shown in fig. 1 comprises a bowl 2 and a screw conveyor 3 which are mounted on a shaft 4 such that they in use can be brought to rotate around a horizontal axis 5 of rotation, the axis 5 of rotation extending in a longitudinal direction of the bowl 2. Further, the rotating body 1 has a radial direction 5a extending perpendicular to the
30 longitudinal direction.

For the sake of simplicity directions "up" and "down" are used herein as referring to a radial direction towards the axis 5 of rotation and away from the axis 5 of rotation, respectively.

The bowl 2 comprises a base plate 6 provided at one longitudi-

nal end of the bowl 2, which base plate 6 has an internal side 7 and an external side 8. The base plate 6 is provided with a number of liquid phase outlet passages 9 having external openings in the external side 8 of the base plate. Furthermore the bowl 2 is at an end opposite to the
5 base plate 6 provided with solid phase discharge openings 10.

The screw conveyor 3 comprises inlet openings 11 for feeding a feed e.g. slurry to the rotating body 1, the slurry comprising a light or liquid phase 12 and a heavy or solid phase 13. During rotation of the rotating body 1 as previously described, separation of the liquid phase 12
10 and solid phase 13 phases is obtained. The liquid phase 12 is discharged through the outlet passages 9 in the base plate 6, while the screw conveyor 3 transports the solid phase 13 towards the solid phase discharge openings 10 through which the solid phase 13 is eventually discharged. As far as described here omitting details relating the solid phase discharge openings 10, the centrifugal separator disclosed in Fig. 1 belongs to the prior art.
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The area around the solid phase discharge openings or outlet openings 10 is shown in more detail in Fig. 2, which shows a narrow end
15 of a conical part 17 of the bowl 2. The narrow end 15 has a radial surface 19 covered by wear resistance members according to a first embodiment of the present invention. An end member 21 of the centrifugal separator has a flange portion 23 with through holes 25 for bolts, not shown, and distance members 27 (cf. Fig. 3) are extending between the radial surface 19 and the flange portion 23. The bolts, not shown, are inserted in the holes 25 and through holes 28 in the distance members 27
20 to be fastened in the conical part 17 thereby mounting the end member 21 on the conical part 17. The distance members 27 may be integral with the conical part 17 or with the end member 21, or they may be separate elements.

30 Between the distance members 27 the outlet openings 10 are present. The outlet openings 10 shown in Fig 2 are fitted with wear resistance members of the first embodiment.

In the first embodiment bushing members 29 has a tubular mantle portion 31 surrounding the respective distance member 27.

Though not circular, the distance members 27 are cylindrical in the shown embodiment and accordingly the tubular mantle portion 31 is also cylindrical to match the distance member. However the distance members and accordingly the tubular mantle portions may be conical, preferably slightly conical.

The tubular mantle portion 31 of the bushing member 29 carries at the end adjacent the radial surface 19 a flange 33 abutting said radial surface.

Between adjacent bushing members 29 plate members 35 are fitted onto the radial surface 19. The plate members have an hourglass-shape with a waist portion 37. Further the adjacent edges 39 of the flanges 33 and the plate members are stepped, whereby the edges of the flanges 33 are overlapping the edges of the plate members 35. Thus the flanges 33 are securing the plate members 35 due to the overlap and the hourglass-shape of the waist portion 37. The bushing members 29 in turn are secured by the fit on the distance members 27 between the radial surface 19 and the flange portion 23. Preferably cement, or a like filler, is used to fill any gaps between the wear resistance members and the adjacent surfaces of the bowl including the end member and especially the distance members. Such filler assist the securing of the wear resistance members and avoid rattling.

The embodiment shown in Fig. 2 further comprises separate corner elements 41 attached by means of glue for the wear protection of the internal edge of the conical part 17 adjacent the radial surface 19.

Figs. 4 to 8 shows a second embodiment of wear resistance members that might be fitted at the outlet openings 10 as an alternative to the bushing members 29 and the plate members 35.

The second embodiment comprises cylindrical bushing members 43 and plate members 45. Like mentioned in relation to the first embodiment, in a variant the bushing members might be more or less conical to correspond to the shape of the distance members 27. The members of the second embodiment being adapted to the same centrifugal separator as the members of the first embodiment, the cross-section of the bushing member 43 of the second embodiment may be identical to

the cross-section of the tubular mantle portion of the bushing member 29 of the first embodiment, as shown in Fig. 3. The bushing member 43 has at one end projections 47a, 47b to be received in recesses, not shown, in the flange portion 23.

5 The plate members 45 are in the second embodiment adapted to form a ring, cf. Fig. 8, covering the radial surface 19. The plate member 45 has an opening 49 shaped to receive the bushing member 43 with a loose fit. Beside the opening 49 the plate member 45 has a hole 51 with a recessed edge 51a for receiving a fastening member such as a
10 bolt (not shown).

 The plate member 45 has a concave edge 53 with an extension 55 rising from the plane of the surface 57 abutting the radial surface 19 when the plate member has been mounted, whereby the extension 55 extends around the internal edge of the conical part 17 adjacent the ra-
15 dial surface 19 to protect said internal edge. Accordingly, said extension 55 is similar to the separate corner elements 41 of the first embodiment.

 At either end of the concave edge 53 the plate member has stepped edge 59, 61, whereby the stepped edges are oppositely stepped so that an edge 59 adjacent the hole 51 of one plate member 45a may
20 overlap an edge 61 adjacent the opening 49 of a neighbouring plate member 45b when the plate members 45 are mounted to form a ring on the radial surface 19.

 Mounting of the plate members 45 is performed as follows: The plate members 45 are assembled into a ring as shown in Fig. 8 on a flat
25 steel ring (not shown). The steel ring comprises openings corresponding to the openings 49 for the distance members 27 to extend through and holes corresponding to the holes 51, but having a smaller diameter. The plate members 45 are fastened to the steel ring by means of e.g. glue and the steel ring is fastened to the radial surface 19 of the narrow end
30 of the bowl of the centrifugal separator by means of bolts with a head so shaped that the bolt is tightened against the rim of the hole in the steel ring without abutting, but only overlaying, the recessed edge 51a of the hole 51 in the plate member. The head of the bolts are accommodated in the holes to be protected from wear. Overlaying the recessed edge 51a

the head of the bolt secures the plate member to prevent its removal in case the glue should fail. The stepped edge 59 of one plate member secured by a bolt through the hole 51 will secure the opposite stepped edge 61 of the neighbouring plate member 45b in case the glue should fail.

Fig. 9 shows an oblique view of four members of a third embodiment, namely two bushing members 63 and two plate members 65. This embodiment combines various features of the first and second embodiments together with some new features. Thus the bushing members 63 comprise a mantle portion 67 with a C-shaped cross-section. When the bushing member has been mounted the mantle portion 67 will surround the distance member sufficiently to prevent the bushing member from being removed in a direction perpendicular to the axial direction.

Moreover, the non-circular cross-section of an inner wall 69 of the bushing member, which corresponds to the cross-section of the distance member, prevents the bushing member 63 from rotating around the distance member. This applies also to the first and second embodiments described above.

The bushing member 63, like the bushing member of the first embodiment, has a flange 71 for abutting the radial surface of the narrow end of the bowl of the centrifugal separator. The plate members 65 and the bushing members have in this third embodiment straight overlapping adjacent edges. Thus the plate member 65 has stepped edges 73 and the bushing member has overlapping stepped edges 75. Thus, when mounted, the bushing members secure the plate members in the axial direction and in the downward radial direction. The bushing member 63 and the plate member 65 further have, at their concave edges, respective extensions 79, 81 rising from the plane of the adjacent surfaces 83, 85, respectively. These extensions 79, 81 correspond to the extensions 55 of the plate members 45 of the second embodiment.

The straight overlapping edges 73 and 75 extend along the ends of the extensions 79, 81, and thus the bushing members 63 also secure the plate members 65 in the upwardly radial direction.

It should be noted that further mixing of features of the three

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embodiments are possible. E.g. in the third embodiment it would be possible to use tubular mantle portions surrounding the distance members completely, like in the first embodiment.

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P A T E N T K R A V

1. Centrifugalseparator omfattende en tromle (2), der under drift roterer i en omdrejningsretning omkring en omdrejningsakse (5), som strækker sig i en længderetning af tromlen, hvilken tromle omfatter et konisk parti (17) med en smal udløbsende (15) omfattende en radialflade (19), et endeelement (21) overfor radialfladen, et antal afstandselementer (27), der strækker sig mellem radialfladen (19) og endeelementet (21) og mellem nabo-afstandselementer (27) tilvejebringer udløbsåbninger (10), hvilke afstandselementer har en aksial udstrækning i omdrejningsaksens (5) aksialretning og en omkredsmæssig udstrækning vinkelret på den aksiale udstrækning, og slidbestandige elementer, som dækker overflader ved udløbsåbningerne, k e n d e t e g n e t ved, at de slidbestandige elementer omfatter et bøsningselement (29; 43; 63) med et kappeparti (31; 43a; 67), som omgiver et respektivt afstandselement (27) i det mindste langs en tilstrækkelig del af den omkredsmæssige udstrækning af nævnte afstandselement til at forhindre fjernelse af bøsningselementet i en retning vinkelret på aksialretningen.

2. Centrifugalseparator ifølge krav 1, kendetegnet ved, at kappepartiet (31; 43a) af bøsningselementet (29; 43) er rørformet og omkredsmæssigt omgiver afstandselementet fuldstændigt.

3. Centrifugalseparator ifølge krav 1, kendetegnet ved, at kappepartiet (67) af bøsningselementet (63) har et C-formet tværsnit vinkelret på aksialretningen.

4. Centrifugalseparator ifølge ethvert af kravene 1-3, k e n d e -
25 t e g n e t ved, at de slidbestandige elementer omfatter en flerhed af
pladeelementer (35; 45; 65), der hver især dækker en del af radialfla-
den (19).

5. Centrifugalseparator ifølge ethvert af kravene 1-4, k e n d e -
t e g n e t ved, at bøsningselementet (29; 63) ved en ende har en flan-
ge (33; 71), som ligger an mod radialfladen (19) og dækker en del af
30 nævnte radialflade (19).

6. Centrifugalseparator ifølge krav 5, kendetegnet ved, at et pladeelement (35; 65) er anbragt på radialfladen mellem nabobøsningselementer (29), hvor kanter (39; 75) af flangerne af bøsningselementer (29) er anbragt på radialfladen mellem pladeelementer (35; 65).

elementerne overlapper kanter (39; 73) af pladeelementerne.

7. Centrifugalseparator ifølge krav 6, k e n d e t e g n e t ved, at pladeelementerne (35) har et taljeparti (37), og at flangerne er indrettet til at indgribe med taljepartiet (37) og sikre pladeelementerne (35) i kraft af overlappet og taljepartiet (37).

8. Centrifugalseparator ifølge ethvert af kravene 1-4, k e n d e t e g n e t ved, at pladeelementerne (45) hver især har en åbning (49) for et afstandselement (27) at strække sig igennem, og at en kant (59) af et pladeelement (45a) overlapper en hosliggende kant (61) af et hosliggende pladeelement (45b).

9. Centrifugalseparator ifølge krav 8, k e n d e t e g n e t ved, at pladeelementet (45) omfatter et hul (51) for et fastgøringselement, og åbningen (49) for et afstandselement er indrettet til at optage en ende af et bøsningselement (43).

10. Centrifugalseparator ifølge ethvert af kravene 4 og 6-9, k e n d e t e g n e t ved, at en konkav kant (53) af de slidbestandige elementer (45; 63, 65), som dækker en del af radialfladen (19), omfatter en forlængelse (55; 79, 81), der hæver sig fra et plan (57; 83, 85) af det slidbestandige element, hvorved forlængelsen (55; 79, 81) er indrettet til at strække sig omkring en indre kant af det koniske parti (17) tilgrænsende radialfladen (19) for at beskytte nævnte indre kant.

11. Centrifugalseparator ifølge ethvert af kravene 1-10, k e n d e t e g n e t ved, at de slidbestandige elementer omfatter wolframkarbid.

12. Centrifugalseparator ifølge ethvert af de foregående krav, k e n d e t e g n e t ved, at afstandselementerne (27), og følgelig kappepartierne (31; 43a; 67) af bøsningselementerne (29; 43; 63), har et ikke-cirkulært tværsnit vinkelret på aksialretningen forhindrende bøsningselementerne (29; 43; 63) i at rotere omkring afstandselementerne (27).

13. Centrifugalseparator ifølge ethvert af de foregående krav, k e n d e t e g n e t ved, at kappepartiet (31; 43a; 67) passer på afstandselementet (27) med en løs pasning, og et fyldmateriale er tilvejebragt for at fylde spalterne mellem kappepartiet (31; 43a; 67) og af-

standselementet (27).

14. Slidbestandigt element (29; 43; 63) til en centrifugalseparator ifølge ethvert af kravene 1-13, k e n d e t e g n e t ved, at være udformet som et bøsningselement (29; 43; 63) med et kappeparti (31; 5 43a; 67) til at omgive et respektivt afstandselement (27) i det mindste langs en tilstrækkelig del af en omkredsmæssig udstrækning af nævnte afstandselement til at forhindre fjernelse af bøsningselementet (29; 43; 63) i en retning vinkelret på en aksialretning.

15. Sæt af slidbestandige elementer til en centrifugalseparator 10 ifølge ethvert af kravene 1-13, omfattende bøsningselementer (29; 43; 63) med kappepartier (31; 43a; 67) til at omgive et respektivt afstands-element (27) i det mindste langs en tilstrækkelig del af en omkredsmæssig udstrækning af nævnte afstandselement til at forhindre fjernelse af bøsningselementet (29; 43; 63) i en retning vinkelret på en aksialret- 15 ning og pladeelementer (35; 45) til at dække et parti af radialfladen mellem nabo-bøsningselementer (29; 43; 63).

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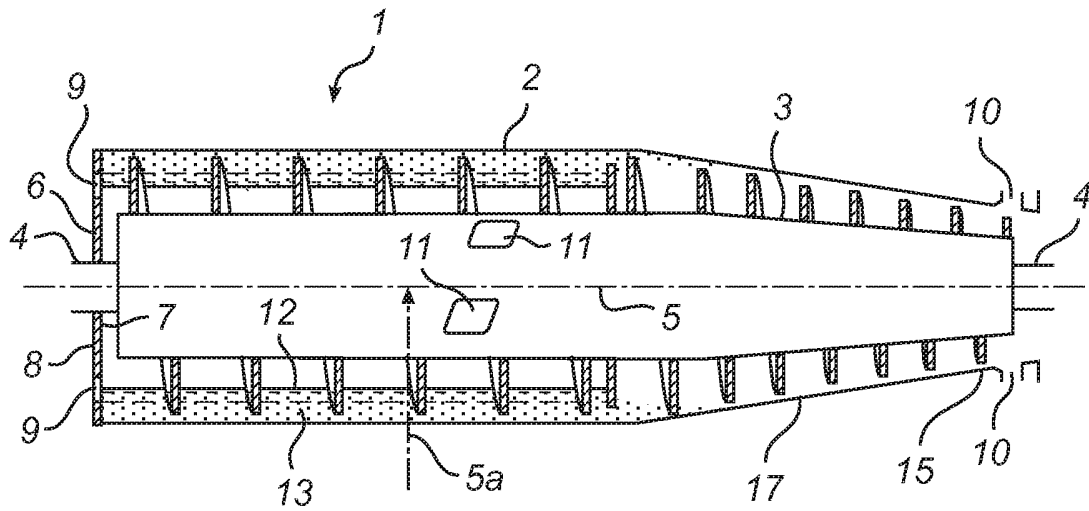


Fig. 1

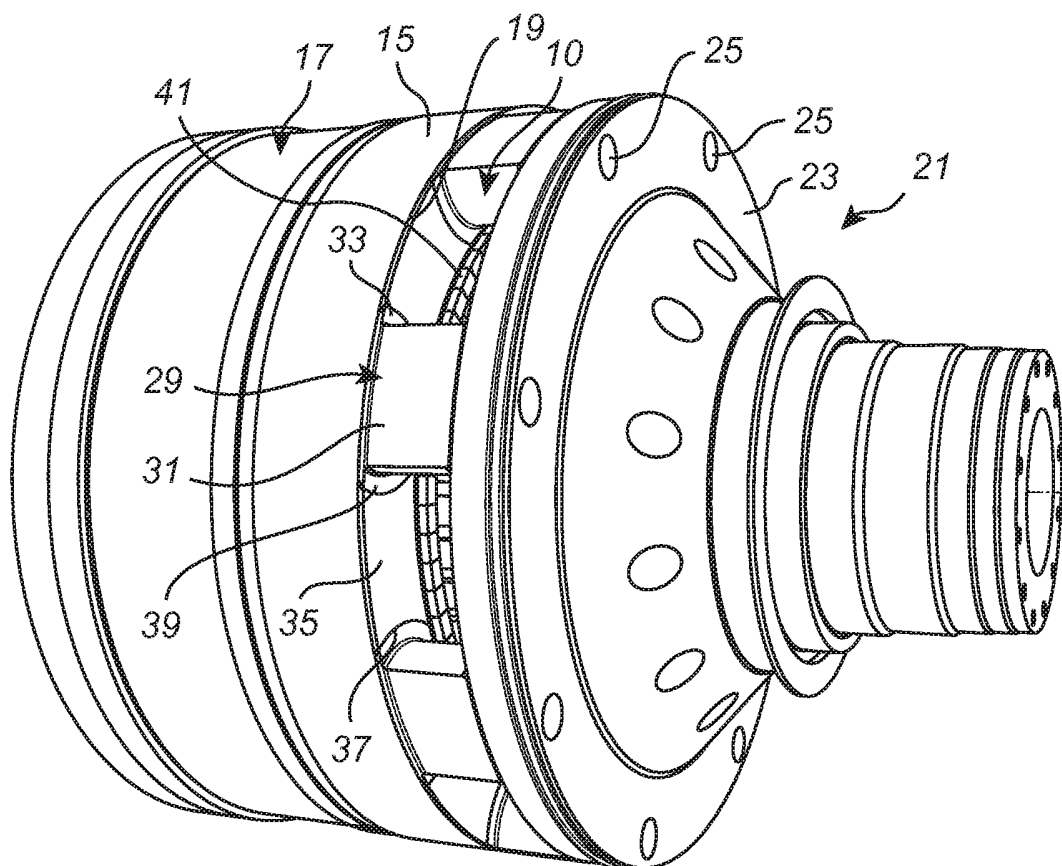


Fig. 2

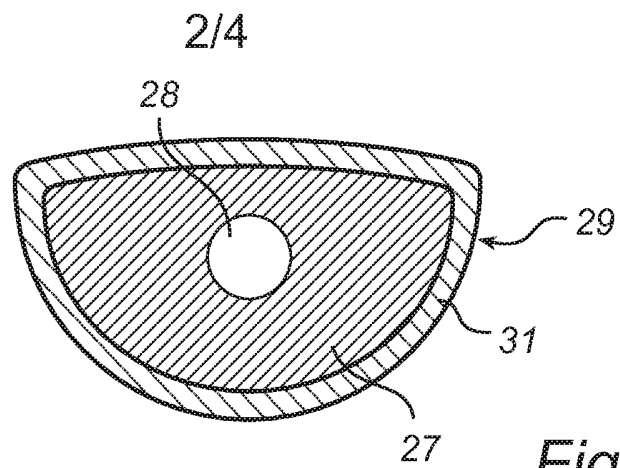


Fig. 3

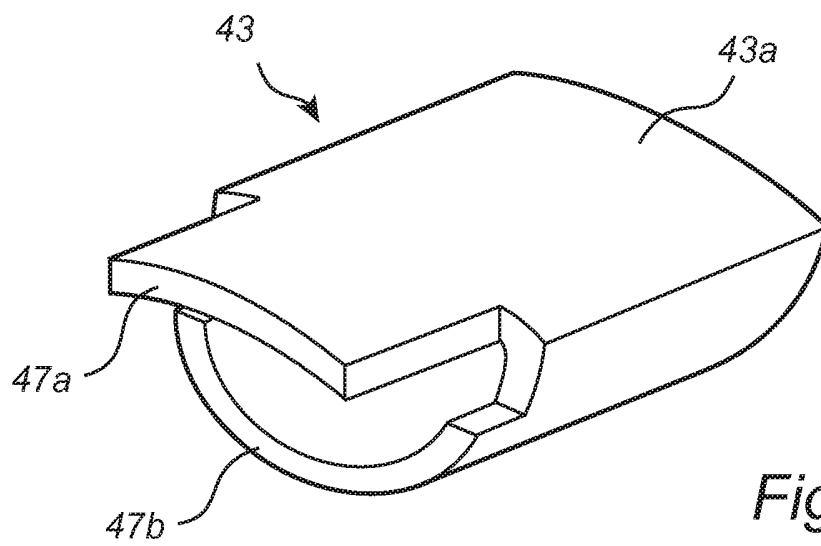


Fig. 4

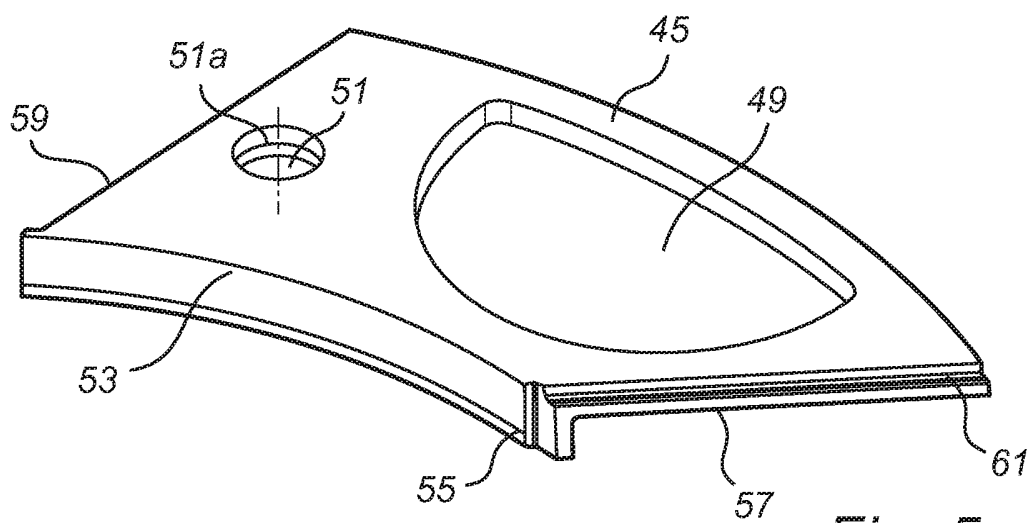


Fig. 5

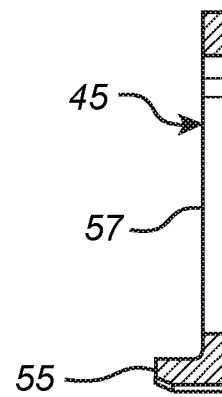
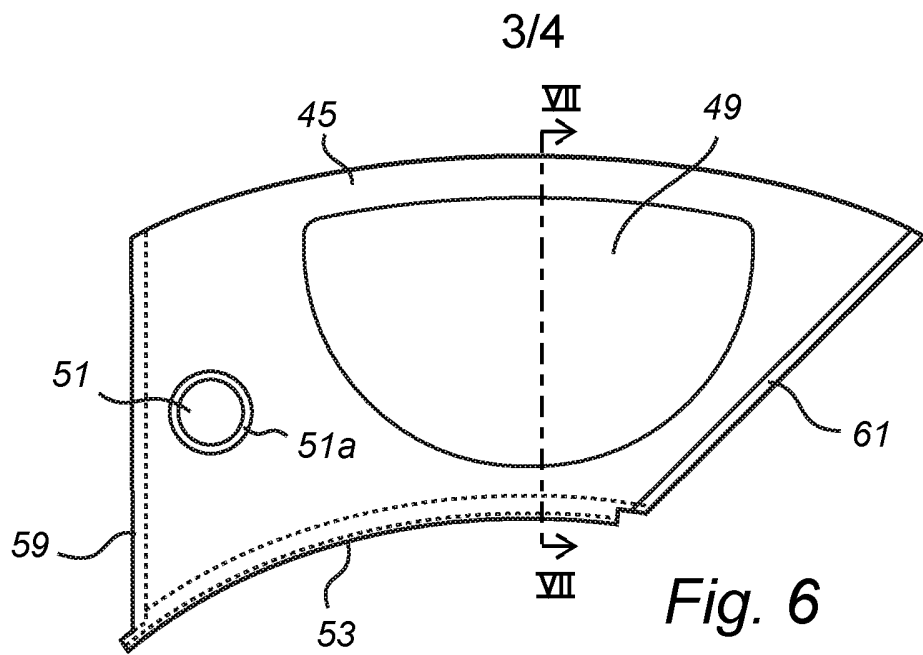


Fig. 7

