

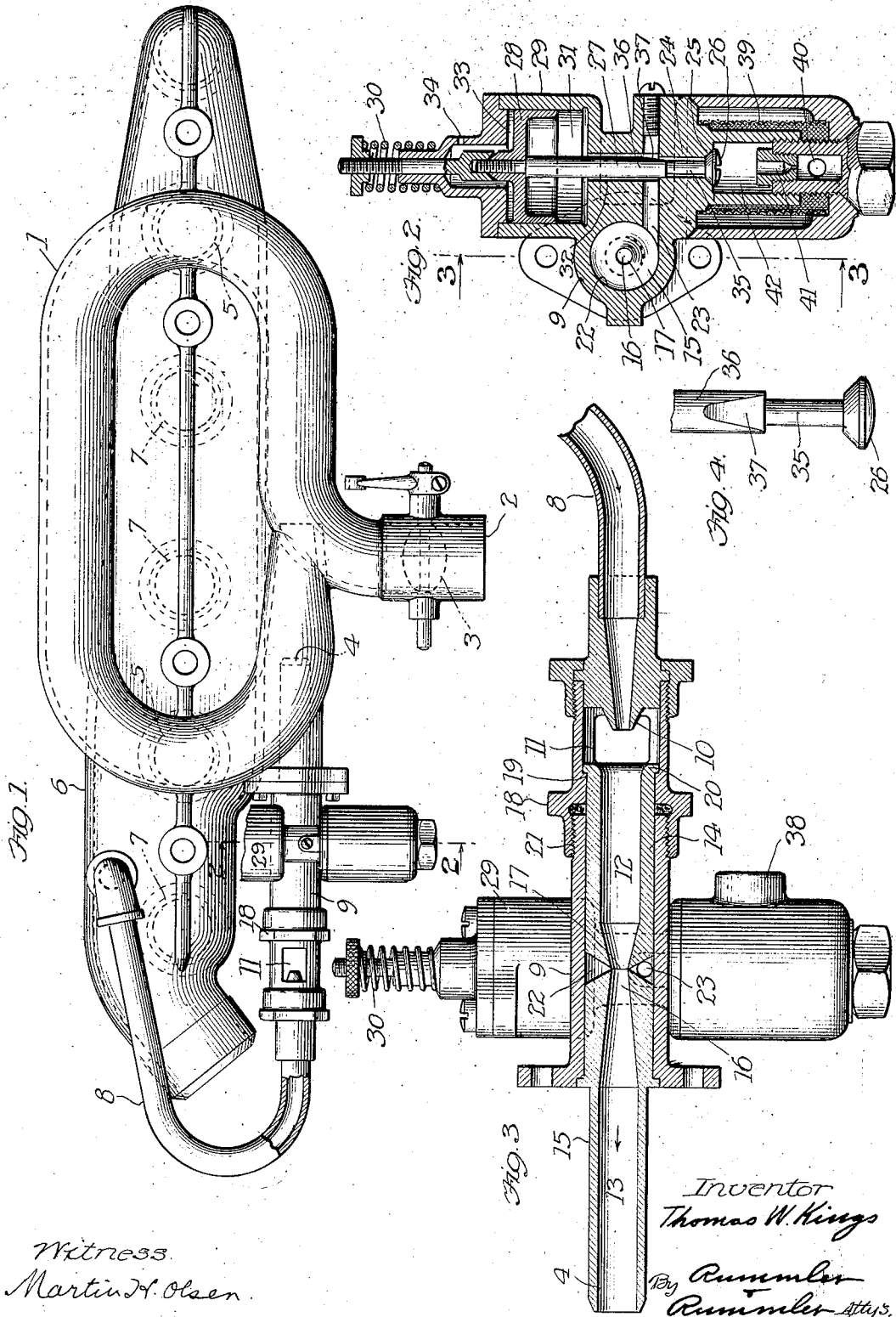
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CARBURETOR

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CARBURETOR.

REISSUED

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The purpose of this invention is to provide a carburetor for governing, and automatically regulating the supply of fuel to, internal combustion engines, as required under different conditions of operation. Thus, when an engine, equipped with this carburetor, is started the carburetor will first operate to admit a maximum quantity of fuel in proportion to the air taken by the engine. As the latter warms up and is operating more effectively and at a speed producing greater suction, the fuel orifice of the carburetor automatically decreases in area and thus reduces the supply of fuel to what is required for idling speeds; or, if the engine is under load, the carburetor will function to supply the proper mixture which is normally required for the most efficient operation of the engine according to the load carried.

The present invention is further distinguishable from prior art carburetors in that when the engine is subjected to an abnormal load and is consequently slowed down to near the starting speed, the fuel supply orifice will again automatically increase in size to compensate for the loss of suction.

It is also the object of the invention to effect a more thorough disintegration of the fuel than is possible with carburetors heretofore in use, particularly when the speed of the engine is decreased due to a heavy load.

It is also among the purposes of the invention to provide a charge-forming device of simplified construction not requiring the use of a float, but which, nevertheless, operates automatically to maintain a fuel level in the carburetor, and to provide a carburetor which insures the required feed of the fuel from a supply tank at lower level, without the use of pressure pumps or the well-known vacuum tanks.

The objects of the invention are accomplished by the device shown in the accompanying drawings, in which:—

Figure 1 is a face view of the engine intake and exhaust manifolds showing partly broken away, the carburetor and its connections to these manifolds.

Figure 2 is a sectional view of the carburetor on the line 2—2 of Figure 1.

Figure 3 is a sectional view taken on the line 3—3 of Figure 2.

Figure 4 is a detail view of the fuel valve in the carburetor.

In fuel feeding and disintegrating devices the disintegration of the fuel is usually proportional to the speed of the air passing the fuel orifice of the disintegrating device. Also, when a thorough mechanical disintegration of the fuel is effected, there is less need for the use of heat in preparing the charge of fuel and air for use in the engine cylinders, and consequently a higher volumetric efficiency of the engine is assured. In carburetors now generally in use, the speed of the air past the fuel orifice does not increase in proportion to the opening of the throttle. With the engine idling, and the throttle nearly closed, there is, of course, a very considerable vacuum in the intake manifold, but it is not available for atomizing the fuel because of being cut off from the fuel by the throttle valve. When the throttle is opened to compensate for increased load or to speed up the engine, although the vacuum in the manifold decreases. The speed through the carburetor will increase because what vacuum there is is available for action in the carburetor, but after the throttle is further opened, the vacuum in the manifold will continue to decrease but the speed through the carburetor will not increase in proportion to the throttle opening, although at this time it is needed most to atomize the greater quantity of fuel which is necessary to pull the increased load. There is also a reduction in the pumping action of the engine because of its lower speed when under load. The degree of vacuum in the intake manifold is proportional to the speed of the engine at any given position of the throttle, but the vacuum is least when the throttle is wide open because of the free inrush of air.

In the present construction, compensation is made for a reduction in vacuum in the intake manifold by the use of a fuel injector, preferably operated by the pressure in the exhaust manifold. The carburetor utilizes a piston under the influence of the partial vacuum in the intake manifold, to regulate the size of the fuel orifice in the carburetor, and to maintain a fuel level in the carburetor. The fuel is, therefore, drawn from the supply tank directly

through the carburetor by the depression in the intake manifold, and the quantity of the fuel delivered to the engine is, therefore, in a measure dependent upon the vacuum in the intake manifold, except as modified by the exhaust operated injector and the position of the throttle, which in this case may merely control the main air supply to the intake manifold. When the loads are light or the engine is idly running, the fuel venturi functions mostly by the partial vacuum in the intake manifold, while with heavier loads and wide open throttles, it functions mostly by the pressure in the exhaust manifold, through the influence of the jet operated by the exhaust gas. The pressure of the exhaust gases increases with the opening of the throttle when the loads are heavy.

The construction as illustrated in the drawings, includes an intake manifold 1 around which the charge may circulate and recirculate, having a main air inlet 2 controlled by a throttle 3, and a fuel and air inlet 4. The outlets 5 of the intake manifold each lead to the intake ports of a pair of cylinders. The exhaust gases from the cylinders are delivered to the exhaust manifold 6 through the openings 7 therein. A conduit 8 leads from the exhaust manifold to the carburetor 9. Conduit 8 terminates at the carburetor in a nozzle 10, at the sides of which are air inlets 11.

With this arrangement, the pressure in the exhaust manifold is utilized to cause a small stream of the exhaust gases at high velocity to be delivered by the nozzle 10 into the passageway 12 in the carburetor, which passageway is in alinement with a passageway 13, terminating in the opening 4 to the intake manifold. The jet of the exhaust gas delivered by nozzle 10 serves to draw air into the carburetor through the openings 11. Air is also urged to flow through these openings due to the reduction in pressure of the intake manifold 1. The passages 12 and 13 are formed in tubular members 14 and 15 with their inner ends tapered to form a Venturi passageway 16.

Fuel enters the Venturi passageway through a small gap 17, at the point of least diameter of the venturi. This gap is regulated as to size by a permanent setting of the tubes 12 and 13, tube 12 being adjustable with respect to tube 13 by means of a nut 18, having a shoulder 20 on tube 12 and having threaded engagement at 21 with the carburetor 9. The space 22 surrounding the adjacent ends of tubes 12 and 13 forming the venturi, communicates through a transverse passageway 23 and a vertical passageway 24 with a fuel port 25 controlled by valve 26. The stem 27 of this valve at its upper end carries a piston 28 which is slidable in a cylindrical part 29 of the carburetor body. A spring 30 normally urges

the piston to the upper position where it holds valve 26 closed. The chamber 31 below the piston is subject to the reduced pressure in the intake manifold and the aspiratory action of venturi 16, since this space is in communication with passageway 23 through the vertical passageway 32 surrounding the stem of valve 27. The space 33 above the piston 28 is in communication with the outer atmosphere through the port 34. The position of the piston is, therefore, dependent upon the difference in pressure between the atmosphere pressure above the piston and the partial vacuum below it. Therefore, when the engine is in operation, valve 26 is lowered, thus permitting fuel to rise in the passageway 24 past the reduced part 35 of the valve stem and into the passageway 23. The upper enlarged part 36 of the valve stem 27 is of the same diameter as the vertical passageway 24, but is tapered at 37 near the upper end of the passageway 24. This tapered part of the valve stem forms a second valve, which, when valve 26 is moved from its seat, operates as a regulating means for automatically controlling the flow of fuel and maintaining the desired fuel level in passageway 23. The fuel supply pipe is connected to the carburetor at the boss 38, Figure 3. The fuel passes through the screen 39, ports 40 and 41 before entering the space 42 below valve 26.

In the operation of the device, a fuel level is maintained in the transverse passageway 23 of the carburetor, or its rate of delivery to this passageway is controlled by means of a partial vacuum in the space 22 surrounding the fuel orifice 17 of venturi 16. When there is a reduced pressure in this space, there is a difference in pressure between the upper and lower sides of piston 28, and the piston is thereby lowered from its normal position against the action of spring 38. The valve 26 is thereby moved from its seat allowing the fuel to be drawn into the passageway 23 and from there through the space 22 surrounding the orifice 17 of the venturi and into the venturi where it becomes disintegrated and mixed with the air entering the carburetor through the openings 11. The rate of flow of the fuel and air is dependent upon this difference in pressure, which in turn depends upon the reduced pressure in manifold 1, according to the speed and pumping action of the engine pistons. However, the reduced pressure in the space 22 surrounding the fuel venturi is also dependent upon the injector action of the exhaust gases passing through the venturi from nozzle 10. The pressure of the exhaust gas is also dependent upon the work performed or load on the engine. Therefore, when there is a decrease in the vacuum in manifold 1 due to a slowing up of the en-

5 engine when under load, this is compensated
 for by the increased injector action due to
 the increase in pressure in the exhaust mani-
 fold. The tapered part 37 of valve stem
 28 insures a maximum supply of fuel when
 valve 26 is first moved from its seat and
 gradually restricts the flow by more or less
 closing the upper end of passageway 24 as
 10 the piston 28 moves downwardly upon an
 increase in suction due to speeding up of the
 engine. Likewise, in the reverse motion of
 piston 28 when there is a falling off in dif-
 ference in pressure below and above the pis-
 ton the flat tapered part 37 of valve stem 27
 15 serves to permit a slight increase in free flow
 of the fuel as valve 26 approaches its seat.
 An excessive flow of fuel through the car-
 buretor is prevented by valve 26 moving to-
 ward its seat whenever passageway 23 be-
 20 comes choked, since closing or partly clos-
 ing passageway 23 cuts off communication
 between the partial vacuum in chamber 22
 and chamber 31 under the piston 28. The
 piston then rises under the action of spring
 25 30, closing or partly closing valve 26. When
 the fuel level is again lowered, communica-
 tion is restored between chamber 31 and the
 intake manifold, causing the piston 28 to
 again move downwardly and open the valve.
 30 Although but one specific embodiment of
 this invention has been herein shown and
 described, it will be understood that numer-

ous details of the construction shown may
 be altered or omitted without departing from
 the spirit of this invention as defined by the
 following claims.

I claim:

1. A charge-forming device of the class
 described comprising a Venturi tube, an in-
 take manifold with which said Venturi tube
 40 is in communication at one of its ends, a
 fuel passageway in communication with said
 Venturi tube near its point of smallest di-
 ameter, a valve for controlling the level of
 fuel in said passageway, and means for op-
 45 erating said valve, said means being sub-
 ject for operation, to the aspiratory action
 of said Venturi tube and the level of the
 liquid in said passageway.

2. A charge-forming device of the class de-
 50 scribed comprising a Venturi tube, an ex-
 haust manifold with which said Venturi
 tube is in communication at one of its ends,
 a fuel passageway in communication with
 said Venturi tube near its point of smallest
 55 diameter, a valve for controlling the level
 of fuel in said passageway, and means for
 operating said valve, said means being sub-
 ject for operation, to the aspiratory action
 of said Venturi tube and the level of the
 60 liquid in said passageway.

Signed at Chicago this 24th day of Feb-
 ruary 1923.

THOMAS W. KINGS.