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(54) APPARATUS AND METHOD FOR  
 TESTING WALLS AND RIMS OF CONTAINERS

(71) We, COORS CONTAINER COMPANY, a Corporation organised and existing under the laws of the State of Colorado, United States of America, of 17755 West 32nd Avenue, Golden, 5 State of Colorado, United States of America, do hereby declare the invention, for which we pray that a patent may be granted to us, and the method by which it is to be performed, to be particularly described in and by the following statement:—

This invention relates to apparatus and methods for testing the walls and rims of containers and, more particularly, to apparatus and methods for testing metallic can body members 15 for defects by the use of light.

The use of light to test containers and container materials has been known since at least 1908 as evidenced by United States Patent No. 901, 393 and many patents have been 20 granted in the field, including United States Patents No. 1, 965,819; 2, 229,451; 2, 246,906; 2,318,856; 2,453,720; 2,481,863; 2,561,406; 2,563,213; 2,682,802; 2,729,136; 2,750,519; 2,872,039; 3,107,011; 3,159,279; 3,171,033; 25 3,327,849; 3,328,000; 3,416,659; 3,453,054; and 3,750,877.

The present invention relates to new and improved apparatus and methods of the general type disclosed in United States Patent 30 No. 3,750,877 in which metallic can body members are carried by a continuously rotating transfer wheel to a defect detection station wherewith the open end or ends of a can body are brought into sealing engagement with sealing 35 means, the exterior surfaces of the can body are flooded with light from a light source; and a light sensing device detects any light passing through the exterior surfaces of the can body to activate reject means to segregate defective 40 can body members from satisfactory can body

members.

Until the present invention, no light testing apparatus has been available which provided completely satisfactory results in terms of high speed accurate testing of both the walls and flange portions of can body members, relatively low cost construction, easy maintenance, and reliability in use with minimum down time for maintenance and repairs. A primary problem in the manufacture of can body members, such as, 45 for example, one piece thin wall aluminium sheet metal can body members, is maintaining defect free flange portions. Small, e.g., .005 or larger, edge cracks in the flange portion may cause leaks in cans during completion of a can by association of end closing members with the can body members by application of relatively high forces to effect closure of the can body members. In addition, it is desirable to detect pin holes at least as small as .001 inch in 50 the walls of the can body members which could cause leaks in use. In the beer industry, some beer manufacturers pasteurize the beer after a can has been filled and sealed, while other manufacturers do not pasteurize the beer. Often times the heat and pressure of the 55 pasteurization process will reveal leaks in the finished cans by causing some of the contents to be driven through the cracks so that leaking cans may be visually observed and removed 60 from the production line. However, if there is no pasteurization process, the leaks may not be detected at the time of manufacture and the leakage may contaminate or spoil adjacent containers or packages while in addition the product may deteriorate or spoil prior to 65 consumption. Thus, in order to maintain high quality standards, it is imperative that all possible defects which may produce leaks be detected prior to completion of the can and 70 75 80

filling of the can with the contents.

In addition, the testing apparatus is arranged and constructed so that damaged portions of the can body member other than the flange which cause the bottom peripheral rim surface to be other than in a flat condition will also be detected and rejected due to loss of vacuum by which the can body members are held during the transfer process.

10 The present invention provides for inspection of can body members with nearly 100% detection of such defects. It is contemplated that the percentage of successful detection of such defects is such that, on the average, approximately only one defective can body member in one million will not be detected and ejected by the testing apparatus. In addition, the present invention provides testing apparatus which is continuously operable at relatively high speeds up to or in excess of 1000 can body members per minute. Furthermore, the cost of manufacture of the testing apparatus has been substantially reduced as compared with apparatus of the type disclosed in United States Patent No. 3,750,877 with maintenance, repair, and down time substantially improved to increase overall reliability. In addition, the present invention provides printing detection means for determining the presence or absence 30 of a printed label on the exterior surface of the can body member.

Another feature of the present invention resides in the provision of new and improved light source means for flooding the exterior surfaces of the can body members with light in the test position without requiring the use of light reflective surfaces on the apparatus as have been used in prior art apparatus. In the present invention a relatively few fluorescent lamps are utilized, such as six to seven, whereas prior art apparatus has employed as many as 19 ultraviolet lamps including the extensive use of reflective apparatus surfaces in order to achieve satisfactory results. Instead of using reflective apparatus surfaces, it has been determined that by proper positioning of a relatively small number of lamps, and in particular fluorescent lamps with associated lamp reflectors and black anodization of surrounding metallic surfaces, exceptionally good results can be obtained.

In addition, the present invention provides uniform light intensity around the entire can body member by particular arrangement and spacing of the lamps. Other features of the present invention include supporting the can body members by vacuum applied to the outside bottom wall, internal pressurization of the can body member during testing, discharge of defective can body members only by inertial forces and gravity after release of vacuum, and removal of acceptable can body members directly from the transfer wheel to gravity discharge chute apparatus.

65 In general, the inventive concepts involve

the use of a new and improved gravity feed system; a new and improved can body flange sealing system with pressurization of the seal and the can body during inspection; a new and improved lighting system; a new and improved mounting of a light detector device relative to a continuously rotating transfer wheel; new and improved ejection apparatus for removal of defective can body members from the transfer wheel; and new and improved transfer apparatus for removing non-defective acceptable can body members from the transfer wheel.

70 The inventive concepts are illustrated in apparatus comprising a presently preferred embodiment thereof on the accompanying drawing in which:

Figure 1 is a side elevational view of the apparatus;

Figure 2 is an end view of the apparatus of Figure 1;

Figure 3 is an enlarged partial cross-sectional side elevational view of the left hand portion of the apparatus of Figure 1 taken along line 3-3 in Figure 2;

Figure 4 is an enlarged partial cross-sectional side elevational view of an intermediate portion of the apparatus of Figure 1 axially next adjacent the apparatus shown in Figure 3;

Figure 5 is an enlarged partial cross-sectional view of the right hand portion of the apparatus of Figure 1 axially next adjacent the apparatus shown in Figure 4;

Figures 6 and 7 are enlarged partial cross-sectional views, with parts removed, taken along the line 6-6 in Figure 1;

Figure 8 is an enlarged cross-sectional view of a portion of the apparatus of Figure 4 enclosed by the dashed line 8-8 prior to sealing association of the can body member with a sealing means;

Figure 9 is an enlarged cross-sectional view of the portion of the apparatus of Figure 4 enclosed by the dashed line 8-8 after sealing association of the can body member with the sealing means;

Figure 10 is an enlarged end view of a portion of the apparatus taken along the line 10-10 in Figure 6;

Figures 11 and 12 are enlarged schematic representations of portions of the apparatus illustrating the sequence of operation thereof;

Figure 13 is an enlarged cross-sectional view of a portion of the sealing means and a portion of the can body member of Figure 9; and

Figure 14 is an enlarged cross-sectional view of a portion of the apparatus of Figure 6.

While certain of the inventive concepts are applicable to apparatus for testing any tubular member for sidewall defects, such as pin holes, or for flange defects, such as cracks, the present invention is particularly adapted for testing of one piece aluminum can body members utilized for manufacture of two piece aluminum cans. As shown in Figures 8 and 9, such

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one piece aluminum can body members comprise an annular sidewall portion 16, closed at one end by an inwardly domed concave bottom wall portion 17 to define a container 5 cavity 18, while having an annular opening 19 at the opposite end surrounded and defined by a formed generally radially outwardly extending annular flange portion 20 having a generally radially extending outer side surface 10 21 terminating in an annular edge portion 22. Such can body members are conventionally utilized to package beer, soft drinks, and other products by filling the container cavity 18 with the product and then sealingly 15 attaching an end closure member over the opening 19 in sealed association with the flange portion 20 which is further deformed during the attachment process to effect a sealed relationship with the end member. In addition, 20 the outer peripheral surface of sidewall portion 16 conventionally has a label of printed ink applied thereto.

It is desirable to inspect such can body members for "defects" prior to filling the can 25 body member with the product to be packaged therein and prior to associating the end closure member therewith. Among the various "defects" in the can body member which should be preferably detected before filling 30 and closing are: (1) Any pin holes which will prevent complete sealing of the contents; (2) any cracks or deformation in the flange portion 20 which will prevent proper sealed association with the end closure member; 35 (3) any dents or deformation in the sidewall portion 16 which may affect the round annular conditions of the flange portion 20 and, hence, prevent proper sealed association with the end closure member or the appearance of the 40 finished filled container; and (4) the absence of a printed ink label on the outer peripheral surface of the sidewall portion 16.

In general, the presently preferred embodiment 45 of the invention shown in the accompanying drawings comprises: frame and housing means 30 having upper and lower portions 32, 34 for the machine components; horizontally extending rotatable drive shaft means 36 for continuous rotation of various machine 50 components about a central axis of rotation 38; electric motor means 39, belt-pulley drive means 40, and speed reducer means 41 for causing continuous rotation of the rotatable shaft means; can body member transfer wheel 55 means 42 mounted on the rotatable shaft means for continuous rotation therewith and for carrying can body members in a circular path 43, Figures 6 and 7, in the direction of the arrows thereabout; a plurality of pocket 60 means 44 circumferentially spaced about the outer periphery of said transfer wheel means for receiving a can body member 14 and supporting the can body member sidewall portion 16 in each pocket means, there being twelve 65 axially spaced pairs of such pocket means in the

illustrative embodiment, and for transferring each can body member along the circular path of movement only during a portion of each revolution of the transfer wheel means with each pocket means being carried generally upwardly between lower vertical center line 45 and upper vertical center line 46 during 180° of each revolution and generally downwardly during the other 180° of each revolution of the transfer wheel means; infeed means 48 non-rotatably mounted on the frame and housing means for loading one can body member in each of the pocket means during the last 90° of the generally upward movement of the pocket means between horizontal center line 49 and upper vertical center line 46 during each revolution; empty pocket detection means 50, Figure 6, for providing a control signal whenever no can is placed in the pocket means 44 at the infeed means 48; non-rotatable seating means 51, mounted in juxtaposition to the infeed means for seating each can body member on a surface of each of the pocket means during the last 90° of the generally upward movement of the pocket means between center lines 49, 46 during each revolution; extendable and retractable means 52, Figure 1, mounted on the drive shaft means for rotation therewith, there being one such means for each of the pocket means, and for axially moving each can body member in each of the pocket means between a first axially retracted position and a second axially extended position during each revolution; releasable holding means 54 associated with each of the axially extendable and retractable means for abutting and releasable holding engagement with the bottom wall portion of each can body member in each of the pocket means during predetermined portions of each revolution; flange portion sealing means 56 associated with each of the axially extendable and retractable means and each of the pocket means and being mounted on the drive shaft means for rotation therewith and for sealable engagement with the flange portion 20 of each can body member carried in each of the pocket means during the first 90° of the generally downward movement of the pocket means between upper vertical center line 46 and horizontal center line 57, Figure 6, during each revolution; pressurization means 58, Figure 3, associated with each of the sealing means for applying pressurized air to the sealing means for obtaining a minimum area of sealable engagement between the sealing means and the flange portion 20 of each can body member and for applying pressurized air through the can body member opening 19 to the container cavity 18 to apply outwardly directed force on the interior surfaces of the can body member to outwardly flex the sidewall portion 16 and the bottom wall portion 17 to enhance the detection of pin 70 75 80 85 90 95 100 105 110 115 120 125 130

holes by passage of light therethrough; non-rotatable light applying means 60, Figure 6, for applying light to the exterior surfaces of each pressurized can body member in each of 5 the pocket means in sealable association with the sealing means and being located to provide a continuous light zone during the first 90° of the generally downward movement of the pocket means during each revolution; light 10 detection means 62, Figure 1, non-rotatably mounted relative to the transfer wheel means and the sealing means and being located for successive axial alignment with each of the pocket means, after sealable association of the 15 flange portion with the sealing means and pressurization of the sealing means and the can body member carried in the pocket means, in the light zone for receiving light only from the light applying means through the can body 20 member and/or between the flange portion and the sealing means, and for generating a first defective can control signal upon receipt of light to provide an indication of a defective can body member; non-printed 25 can body member detection means 64, Figure 6, non-rotatably mounted relative to the transfer wheel means and located in juxtaposition to the sealing means and adjacent the outer peripheral surface of the sidewall 30 portion 16 of each can body member while being carried by the pocket means along the circular path, for providing a second defective can control signal in response to light reflected from the outer peripheral 35 surface of non-printed can body members; defective can body member discharge chute means 66, Figure 7, for receiving defective can body members from the pocket means on the transfer wheel means during only the 40 last 90° of the generally downward movement of the pocket means between horizontal center line 57 and lower vertical center line 45 during each revolution; and non-defective can body member unloading chute means 68, 45 Figure 7, for receiving non-defective can body members from the pocket means on the transfer wheel means during only the last part of the last 90° of the generally downward movement of the pocket means during 50 each revolution.

Referring now to Figures 1 and 2, in general, the test apparatus is mounted in relatively compact frame and housing means 30 comprising a lower motor and control housing 55 portion 34 and an upper test apparatus portion 32. In the illustrative embodiment the housing means has a height of 51 inches, a length of 46 inches, and a width of 29 inches. The conventional electric motor-transmission 60 means 39 is drivably connected by the conventional belt-pulley means 40, and the conventional speed reducer box 41 to the centrally axially extending shaft means 36 rotatably supported by suitable bearing means 65 70, 72, Figures 3-5.

The rotatable can body member transfer wheel means 42 is fixedly mounted on a central portion of shaft means 36 for continuous rotation therewith in a closed or partially closed generally annular test chamber 74 having 70 an access opening 76, Figure 2, closable by a light sealing door means 78 pivotally mounted at 80, Figure 2.

The transfer wheel means 42 comprises a pair of axially spaced annular transparent 75 plate members 82, 84 on which the plurality of peripheral circumferentially spaced and axially aligned can body receiving pocket means 44, Figures 6 and 7, are provided. A hub member 85 fixedly mounts the plate members 82, 84 on shaft means 36 for continuous rotation therewith.

The sealing means 56 are rotatably carried by an annular sealing wheel means 86, and peripherally mounted thereon in circumferentially spaced coaxially aligned relationship with pockets 44. Wheel means 86 is fixedly mounted on shaft means 36 by hub member 85 for continuous rotation therewith and located in axially spaced relationship to transfer wheel 90 member 82.

The light detector means 62 is fixedly mounted on the side wall of chamber 74 and extends axially into chamber 74 with a sealing head means portion 88 coaxially alignable with sealing means 56 and mounted in sealed engagement with the adjacent side surface of sealing wheel means 86.

The releasable holding means 54 are coaxially mounted on the ends of the extendable and retractable means 52 in circumferentially spaced and coaxial alignment with pockets 44. The releasable holding means 54 and extendable and retractable means 52 are fixedly mounted on shaft means 36 by a hub member 89 for continuous rotation therewith in axially spaced relationship to transfer wheel member 84. A cam plate means 90, Figure 5, is fixedly mounted relative to shaft means 36 for camming engagement with cam follower means 92 to extend and retract means 52. A can body member guide plate means 94 is fixedly mounted on hub member 89 for continuous rotation with shaft means 36 and has a plurality of circumferentially spaced openings 96, Figure 115 8, coaxially aligned with holding means 54 to enable axial movement of the holding means therethrough.

#### Transfer Wheel Means and Pocket Means

Referring now to Figures 3 and 4, the plate members 82, 84 of the transfer wheel means 42 are of identical construction and each may be made of one piece or mating semi-cylindrical 125 segments, of suitable transparent material, such as Plexiglas (Registered Trade Mark), or Lucite (Registered Trade Mark), to enable uniform transmission of light to the entire outer surface of the can body members during 130

testing. The plate members 82, 84 are fixedly attached to hub member 85 by a plurality of suitable fastening members 100, 102 and hub member 85 is fixedly attached to shaft 5 means 36 by a key member 104.

In the presently preferred and illustrative embodiment, each of the plate members 82, 84 has twelve pockets 44 which are equally circumferentially spaced and 10 coaxial with the pockets of the other plate member. As shown in Figure 7, each pocket comprises a radially formed arcuate segment surface 106 having a center of curvature located at 107 in general coaxial alignment with the 15 holding means 54 and the sealing means 56 so that can body members supported on surface 106 will also be located in general coaxial alignment with the holding means 54 and the sealing means 56. Surfaces 106 extend 20 circumferentially approximately 140° with a trailing surface portion 108 of approximately 95° on the trailing side of a radial line 110, extending from the central axis of shaft 36, and a leading portion 112 of approximately 45° 25 on the leading side of radial line 110. The trailing surface portion 108 merges tangentially with a radial surface 114 of a radially outermost cam lobe portion 116 of the plate members. The leading surface portion 112 30 intersects an elongated curved surface portion 118 to provide a slight transition shoulder at 120. The cam lobe portion 116 extends radially outwardly beyond center 107 so that surface 114 is effective at the can body 35 member loading position adjacent the infeed means 48, Figure 6, to remove the can body member being loaded into the pocket 44 while also providing a cam surface means effective to hold the next can body member in the 40 infeed means 48 clear until the following ramp surface 118 of the following pocket begins to engage the next can body member. Ramp surface 118 provides a cam surface means effective to gradually advance the next can body member to the following pocket. 45 Shoulder 120 is effective as the can body member leaves the loading position and approaches the seating means 51 to properly seat the can body member on surface 106 50 between surfaces 114 and 118.

#### Infeed Means

Referring now to Figures 6 and 10, the 55 infeed means 48 is of the upwardly inclined gravity feed guide chute type having a rectangular passage 130 generally corresponding to the shape of the can body members 14, and connected to a continuous supply of can body 60 members by conveyor means (not shown), by an attachment bracket 131 comprising side plate members 132, 134, and upper and lower plate members 136, 138. Bracket 131 is fixedly mounted on an attachment plate portion 139 65 of a bracket member 140 attached to the

housing and frame means 32. The can body members are rollably, slidably, guidably supported by a pair of spaced elongated adjustable guide plate members 142, 144, lower round spaced elongated guide rail members 146, 148, and upper round spaced elongated guide rail members 150, 152, which are supported at the upper end by bracket 131 and by a downward spaced bracket 153 fixedly supported on plate 139. The lower ends of upper and lower rail members 146, 148, 150, 152, terminate at 154, 155, respectively, in radially outwardly spaced relationship to the plate members 82, 84. The lower end portions of guide plates 142, 144 terminate at 156 and are inclined as shown in Figure 6. The can body members are guided from the lower end of the passage 130 into the pockets 44 by a second guide rail means, in the form of more closely spaced lower round guide rail members 158, 160 fixedly mounted on a cross plate 161 attached to and extending between rail members 146, 148. The lower terminal portions 162, 164 of rail members 158, 160 extend inwardly between and are axially spaced from the transparent plate members 82, 84 and are upwardly curved to provide upwardly curved terminal surfaces 166 extending generally in the direction of rotation of plates 82, 84 and generally tangentially to the surfaces 106 of the pockets 44. As shown in Figure 11, the central longitudinal axis 167 of the chute passage 130 is inclined at an angle of about 25° relative to a horizontal plane and has a common point of intersection 168 with line 43 and a radial line 169 extending from axis 38 at an angle of about 40°. The arrangement is such that can body members are rollably slidably supported in stacked abutting relationship in passage 130 and continuously fed into the rotational path of plate members 82, 84 to continuously load can body members into the pockets 44 as each set of empty pockets rotate past the guide rods 162, 164. An upper guide plate 170, Figures 6 and 10, is mounted on plate 139 opposite rail members 158, 160, and extends inwardly between plate members 82, 84 to provide a downwardly facing guide surface 171 terminating in a curved lower surface 172 extending generally tangentially to the cylindrical upper surface of the can body members, to further guide the can body members into proper position in the pockets 44 as the can body members are removed from the chute. 115

In order to provide a positive stop to selectively prevent can body members from moving downwardly through infeed means 48, a pair of oppositely aligned selectively operable holding and release means 174, 175 may be provided in the form of an extendable and retractable pin members 176, Figure 10, actuatable by solenoids between a retracted release position and an extended holding position in 120 125 130

the chute passage 130 in abutting retaining engagement with a lowermost can body member as shown in Figure 11.

### 5      Empty Pocket Detection Means

In order to provide a control signal indicating the absence of a can body member in pockets 44 after passing the infeed means, 10 an empty pocket detection means 50, Figure 6, in the form of a conventional magnetic field type sensor device 180 mounted in a support member 182 is connected to control means (not shown) by an electrical line 184. Member 15 182 is adjustably fixedly mounted on a bracket member 186 fixed to a fixed plate 188 extending inwardly between rotating plate members 82, 84. The end surface 190 of member 182 is located in a plane closely adjacent 20 to the radially innermost surfaces 106 of pockets 44 so as to be located adjacent and effective relative to the adjacent outer cylindrical surface of the can body members.

### 25      Seating Means

In order to assure positive positioning and location of the can body member in pockets 44 as it is carried from the loading position toward 30 the testing position, a seating means 51 is provided arcuately opposite the pockets 44 along an arcuate segment of the upward path of movement of the can body member beginning at or just beyond the discharge opening defined 35 by surfaces 166, 172 and terminating prior to the vertically uppermost position along center line 46. The seating means 51 is in the form of a pair of spaced arcuate brush segments 200, 201, Figure 10, each of which has a multitude 40 of resilient flexible bristle members mounted on an arcuate backing plate member 202 and extending generally radially inwardly therefrom. The bristle members terminate radially inwardly along an arcuately curved resilient 45 flexible can body member engaging surface 204 located radially outwardly from pocket surfaces 106 a distance such as to firmly engage a substantial arcuate segment of the radially outermost outer peripheral surface of 50 the sidewall portion 16 of each can body member carried thereby. The lower end portions 206 of the bristle members are located in cooperative relationship with the pocket surface 106 and the curved guide surface 172 so 55 as to become effective by engagement with the can body member at about the same time as the can body member is removed from the curved surfaces 166. The resilient flexible guide means thus provided are fixedly mounted 60 relative to the rotating plate members 82, 84 by an arcuately curved portion 207 of bracket member 208 which may be adjustably mounted on the top of plate 139 by suitable fastening means 209 to assure proper alignment and positioning relative to the pockets 44.

### Releasable Holding Means

Referring now to Figures 4 and 8, the releasable holding means 54 comprise twelve separate equally circumferentially spaced axially extendable and retractable units which are rotatable with wheel members 82, 84 in general axial alignment with pockets 44. Each unit comprises an annular mounting block member 210, a spacer member 212, and an annular transparent support plate member 214 attached to member 210 by suitable recessed fastening means 216. A vacuum and air passage 218 extends axially through flat front end surface 220 and is connected to 70 vacuum supply coupling 222 through a passage 224 and a chamber 226 without use of a flap valve to control vacuum conditions as described in United States patent No. 3,370,877. The arrangement, as hereinafter described in further detail, being such that, as shown in Figure 8, the bottom end wall portion 17 of a can body member 14 is held against the surface 220 of plate member 214 with abutting substantially sealing engagement established along the annular rim portion 230 of the can body member to provide a vacuum chamber 234, between surface 220 and the inwardly domed bottom end wall portion 17 connectable to a vacuum source through passage 218. 75

The passages 218, 224, and chamber 226 are connectable to a conventional vacuum source, such as a vacuum pump and control assembly 236, Figure 1, at predetermined times during each revolution, through coupling 222, a flexible hose 238, a coupling 240, Figure 5, a radially extending passage 242 in an annular connecting plate 244 fixed to hub member 89 by suitable fastening members 246 for rotation therewith, an axially extending passage 248, and an arcuate vacuum supply chamber 250 in a non-rotating manifold ring member 252 which is connected to the vacuum source in a manner to be hereinafter described. Pin members 260 are fixedly mounted on an annular ring plate member 262, fixedly non-rotatably mounted on plate member 90 by suitable fastening elements 264, and are loosely received in axially aligned bores 266 in manifold ring member 252 to retain the ring member 252 in non-rotational relationship relative to rotating connecting plate member 244. The bores 266 are larger than the pin members 260 to enable relative axial sliding movement therebetween so that compression spring members 268, mounted circumjacent pin members 260 between axially spaced side surfaces 270, 272, are effective to axially bias the smooth polished side surface 274 of manifold ring member 252 into abutting sealing 100 engagement with the smooth polished side surface 276 of connecting plate member 244. 105

Referring now to Figures 11 and 12, each of the passages 218 of Figure 4 in each of the support head plates 214 is connectable, at 110

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predetermined locations during each revolution, when the associated passage 248 in connecting ring member 244 becomes aligned with the arcuate vacuum chamber 250 which extends 5 arcuately circumferentially approximately 170° in manifold plate member 252 from approximately 6° (at radial line 280) before upper vertical center line 46 to approximately 16° (at radial line 282) before lower vertical 10 center line 45. The vacuum chamber 250 is continuously connected to the vacuum source 236 by a fixedly mounted coupling and a flexible conduit (not shown) extending axially through an axially extending bore in plate 15 member 90 in a manner to be hereinafter described in reference to pressurized air supply means.

The passages 218, 224, 228 are also connectable to a conventional source of pressurized 20 air (not shown) through coupling 222, flexible hose 238, coupling 240, radially extending passage 242, and an axially extending passage 286 which is connectable, at predetermined times during each revolution, to a second 25 axially extending air passage 288 located in non-rotating manifold ring 252 approximately 75° (at radial line 290) below horizontal center line 57. Air passage 288 is controllably connected to a conventional source of pressurized air 30 (not shown) through conventional control valve apparatus (not shown) by a separate coupling member 292, Figure 5, and flexible conduit 294 extending through a bore 296 in non-rotating plate member 90, the vacuum 35 chamber 250 being connected to the vacuum source by a similar arrangement.

#### Extendable and Retractable Means

40 Referring now to Figures 4 and 5, the extendable and retractable means 52 comprise twelve equally circumferentially spaced axially slidable support shaft members 300 coaxially aligned with the pockets 44 and the sealing means 56, there being one of said shaft members supporting each one of said holding means 54. Each of the shaft members are slidably mounted in a housing member 302 and a bearing sleeve 304 and are slidably non-rotatably held on the outer periphery of hub member 89 for rotation therewith by key members 306.

As shown in Figure 5, each of the support shaft members 300 are axially movable between 55 extended and retracted positions by the cam follower means 92 which connect the rear end portions of members 300 through supporting connecting flange members 308 on which are mounted cam follower roller members 310, 312 60 controllably engaged with opposite radially extending annular cam surfaces 314, 316 on the fixedly mounted cam plate means 90. Cam surfaces 314, 316 are contoured to axially extend and retract the support head means 54 65 at predetermined times during each revolution

so as to engage and disengage the can body members relative to the sealing head means 56 as hereinafter described in further detail.

#### Sealing Means

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Referring now to Figures 8 and 9, sealing means 56 comprises twelve circumferentially spaced units mounted on rotatable plate member 86 in coaxial alignment with the pocket means 44 and the holding means 54. As shown in Figure 9, each sealing head means 56 comprises can body member flange sealing means in the form of an annular mounting ring member 320 suitably fixedly connected to 80 the rotatable plate 86 within an annular counter bore 322 by suitable fastening elements 324 and within an annular opening 325 in a ring member 326 mounted on the side surface 327 of member 86 circumjacent mounting ring 85 members 320 by suitable fastening elements 328.

An annular resilient flexible sealing ring member 330 of the general type disclosed in United States patent No. 3,672,208 is fixedly 90 mounted between surface 332 of ring member 320 and surface 334 of plate member 86. Ring member 320 has a radially inwardly extending flange portion 336 with an inwardly tapered front surface 338. Ring member 330, which is made of suitable resilient molded plastic material, such as Neoprene or Urethane or the like, has a U-shaped cross-sectional configuration including first and second identical radially inwardly extending axially spaced annular flange portions 340, 342 connected by axially extending rim portion 344 so as to be reversible. The rim portion 344 is abuttingly sealingly received on annular surface 345 of member 320. The side surface of the inner flange portion is abuttingly received on surface 334 of member 86. The outer flange portion 342 freely extends generally radially inwardly from rim portion 344 for resilient flexible displacement relative thereto. Flange portions 340, 342 extend radially inwardly substantially beyond the annular outer edge portion 22 of the can body members supported in pockets 44 and terminate in an annular lip portion 346 having a diameter less than the outside diameter 115 of the can body member rim portion 22. Thus, the outer side surface 348 of the outer flange portion 342 is engageable with the outer edge 22 of the flange portion along a relatively small width annular portion of the outer surface 348 of portion 342, Figure 13. The arrangement is such as to effect sealing engagement therebetween by substantially line contact when the flange portion of the can body member is in engagement therewith. As shown in Figure 9, during sealing engagement with the rim portion 22, the flange portion 343 is resiliently flexible outwardly bowed so that the outer side surface 348 has a curvature to further assure the desired sealing engagement 130

by substantially line contact. A pressurization chamber 349 is defined by the flange portions 340, 342 and rim portion 344 whereby pressurized air in chamber 349 may be applied to the 5 inner surface 350 of the outer flange portion 342 to further control and obtain the desired sealing engagement with the can body rim portion 22.

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#### Pressurization Means

In order to supply pressurized air to the inside 18 of the can body member and to the chamber 349 of the sealing ring member 15 330 as well as permit passage of light to the light detection means 62, counter bores 351, 352 extend through plate member 86 in coaxial alignment with pockets 44 and sealing means 56. An annular plate 353 of transparent 20 material, such as Plexiglas or the like, is fixed in bore 351 against a shoulder 354 and sealably mounted therein by an O-ring peripheral sealing member 355 to define a chamber 358 to receive pressurized air from a radially extending 25 passage 360 and to enable the passage of light therethrough from chamber 358 to bore 352. With a can body member mounted on the releasable holding means 54 and with shaft member 300 extended to engage the rim 30 portion 22 of the can body member with the sealing lip portion 342, a closed pressure chamber means is defined by the transparent annular plate 353, bore portion 351, the sealing ring means 330, and the can body member 14.

35 In order to pressurize the pressure chamber means at predetermined times, each bore portion 351 is connectable by a radially inwardly extending passage 360 in member 86 to an axially extending passage 362 or 363, 40 Figures 3 and 11, in member 86, which open through side surface 364. A manifold arcuate segment member 366, Figure 3, having two radially offset circumferentially extending arcuate pressure chambers 368, 370, is held 45 in sealing abutting engagement on surface 364 by spring means 372 mounted circumjacent retaining pin members 374 carried by a non-rotatable support plate 376 fixed to another non-rotatable support plate member 378 by 50 suitable fastening elements 380 and spacer elements 382. The arrangement is such that alternate ones of the pressure chamber means 358 are connected to pressure chambers 368, 370 by varying the length of alternate passages 55 360 and the radial location of alternate passages 362 and 363 as illustrated in Figure 11. In this manner, a source of air pressure (not shown) at, for example, between 10 psi and 15 psi is connected by suitable passage 60 means (not shown) to manifold chambers 368, 370 and at predetermined times passages 362, 363 are aligned with chambers 368, 370 to deliver air pressure to the pressure chamber means for purposes to be hereinafter described. 65 In addition, at predetermined times the

passages 362, 363 are located in circumferentially spaced relationship to the manifold member 366, which is in the form of an arcuate segment, whereat the pressure chamber means is vented to the atmosphere. As shown 70 in Figure 11, chambers 368, 370 extend circumferentially approximately 55° from a position, represented by radial line 280 prior to the upper vertical center line 46 to a position slightly beyond the test station represented by 75 radial line 383. It is noted that a portion 384 of axial passage 360 is extended radially outwardly beyond chamber 358 for manufacturing purposes and is suitably sealed as by a threaded closure. 80

#### Light Detector Means

Referring now to Figure 3, the light detector means 62 comprises a conventional photo-multiplier tube assembly (not shown) suitably connected to the control circuitry of the testing apparatus (not shown) for purposes to be hereinafter described. 85

A tubular support housing 385 is non-rotatably adjustably mounted in a bore 386 in support plate member 378 in coaxial alignment with sealing means 56 by a mounting plate 387 and suitable fastening elements 388. A central bore 389 is adapted to receive and support the conventional photo-multiplier tube assembly therewithin. An access cover 390 is provided at one end of the housing 385 and an annular attachment flange portion 392 is provided at the other end. An annular mounting plate 394, having an annular bearing ring member 396 suitably fixedly attached thereto, is fixedly mounted on flange portion 392 by suitable fastening elements 398. Connecting coaxial bores 400, 402 provide light passages 100 and the interface of members 392, 394 is sealed against passage of light by a sealing ring member 404. The bearing ring member 396 is made of good bearing polymer plastic material and has a side surface 406 adapted to slidably 105 sealably engage the adjacent side surface 364 of member 86 thereby preventing the passage of light into housing 385 except through transparent window plate 353 from chamber 358 in member 86. The support housing 385, mounting plate 394, and bearing ring member 396 are axially resiliently biased toward surface 364 by compression springs 407 mounted on pin members 408 adjustably supported on plate 387 by suitable fastening elements 409. 110 115 120

#### Light Applying Means

Referring now to Figures 6 and 14, in the presently preferred embodiment, the light applying means 60, located at the test station, comprise seven elongated fluorescent lamps 412, 413, 414, 415, 416, 417, 418. A first pair of axially extending lamps 412, 413 and associated reflector device 420 are mounted on the 125 130

cover member 78 approximately 30° on one side of the test station and a second pair of axially extending lamps 414, 415 and associated reflector device 422 are mounted 5 on cover member 78 on the other side of the test station. A single outer center lamp 416 and associated reflector device 424, extending generally tangentially to the path of movement of the transfer wheel means, are mounted on 10 cover member 78 directly above the test station. A pair of inner center lamps 417, 418 and associated reflector device 426, extending generally tangentially to the path of movement of the transfer wheel means, are mounted on 15 a bracket assembly 428 non-rotatably supported on the end portion 429 of bracket 188 directly below the test station. It has been determined that fluorescent lamps, such as arranged in Figures 6 and 14, provide a 20 substantially uniform illumination intensity pattern about the can body at the test station without requiring the use of reflectorized machine surfaces as disclosed in prior art apparatus. In fact, all surrounding metallic 25 machine surfaces are anodized to provide black non-reflecting coloring which provides more uniform test results. As shown in Figure 6, each of the axially extending reflector devices 420, 422 are arranged to specifically 30 direct a portion of the light from the associated lamps at the radially innermost half area 429 of the can body member. As shown in Figure 14, each of the circumferentially tangentially extending reflector devices 424, 426 35 are arranged to direct a portion of the light from the associated lamps at both the flange end of the can body member in the general annular area 430 and to a lesser extent at the bottom end of the can body member in the 40 area 431, with there being a high concentration of reflected light directed at the specific annular area 432 of engagement of the rim portion 22 with the sealing lip portion 342. Thus, any pinhole or crack type defects in the 45 can body member will transmit light into the inside 18 of the can body member 14 where any such transmitted light will be detected by the photo-multiplier tube through chamber 358 and window 353. In addition, any cracks in 50 the flange portion 20 of the can body member will permit transmission of light into chamber 358 and any substantial dents will permit passage of light between the rim portion 22 and the sealing lip portion 342 into the 55 chamber 358 for detection by the photo-multiplier tube.

#### Discharge Chute Means

60 Referring now to Figures 2, 4 and 7, the discharge chute means 66 comprises vertically extending axially spaced side plate members 436, 437, an outer vertically extending end plate member 438, which may be removed or 65 further outwardly spaced if desired, and an

inner inclined vertically extending end plate member 439 which define a vertically downwardly extending chute passage 440. Side plate members 436, 437 are axially spaced apart a distance slightly larger than the axial length of the can body members, as illustrated in Figure 4, and are located axially outwardly of the rotating transparent plate members 82, 84 in substantial vertical alignment with the inner side surfaces 441, 442, respectively, 70 of ring member 326 and plate member 94. As shown in Figure 7, the uppermost portion 443 of side plate member 436 terminates about 7° below horizontal center line 57 with an inner portion 444 located radially inwardly of the pockets 44 and an outer portion 445 spaced radially outwardly of the transparent plate members a distance sufficient to enable passage 75 of the can body members thereby. A curved upper surface 446 of side plate 436 extends circumferentially downwardly in radially inwardly spaced and tangential relationship to pocket surfaces 106. The uppermost surface 447, Figure 2, of side plate member 437 is downwardly curved and extends substantially tangentially to the peripheral surface of guide plate member 94. The upper portion 448, Figure 2, of inner end plate member 439 terminates radially outwardly of the 80 transparent plate members 82, 84 below the upper end of the unloading chute means 68. Thus, a circumferentially extending defective can body member inlet opening is defined at the upper end of the discharge chute means 85 between fixed side plate 436 and rotating plate 94. An outlet opening 449, Figure 4, at the bottom of the passage 440 is connected to an enlarged chute passage 450 defined in part by an outwardly curved discharge chute portion 451 in the lower housing portion 34 and having 90 a terminal portion 452, Figure 2, extending beyond the housing portion for placement of defective can body members in a collection bin (not shown). 95

#### Unloading Chute Means

Referring now to Figures 4, 7 and 12, the unloading chute means 68 comprises a generally horizontally extending downwardly inclined (e.g., approximately 15°) elongated rectangular chute passage 454 extending from an inlet opening 455 adjacent plate members 436, 437, 439 of chute means 66 to an outlet opening 456. The lower end of the chute passage 454 is constructed similarly to the upper end of the infeed chute means 48 of Figure 10 and comprises: an attachment bracket 457 for connection to gravity type conveyor means (not shown) for removal of can body members; 115 a pair of spaced elongated adjustable guide plate members 458, 459; a first pair of lower round spaced elongated guide rail members, one of which is shown at 460, Figure 7; a pair of upper round elongated guide rail members, 120 130

461, 462, Figure 4; and a support bracket 463 which are mounted on a bracket 464 fixed to the housing means 32. The first pair of lower guide rail members 460 terminate at 466 5 and the upper guide rail members 461, 462 terminate at 468.

A second pair of lower round spaced elongated guide rail members 470, 471, are coaxially aligned with and form an extension of 10 guide rail members 460. Guide rail members 470, 471 extend between and are fixedly mounted at one end on a transversely downwardly extending plate member 472, which extends therebetween, and at the other 15 end on a pivotally movable hinged cross plate 473 pivotally connected at 474 to a fixed hinge cross plate 475 fixedly mounted across rail members 460 on bracket 464 to provide hinge means for enabling pivotal 20 movement of guide rail members 470, 471 and end plate 472, as indicated by arrow 476 between a normal upper transfer position (shown in Figure 7) forming an extension of rail members 460 and a downwardly displaced 25 position (not shown). A tension spring means 478, connected at one end to a shaft member 479 fixed to and extending across rail members 470, 471 and at the other end to adjustment bolt member 480 attached to bracket 30 463, resiliently biases the rail members 470, 471 and end plate 472 toward the upper transfer position while permitting pivotal downward movement by downwardly directed forces as may be encountered in the event of a 35 malfunction (e.g., jamming) during the unloading of can body members. The first and second pairs of axially spaced lower guide rail members 460, 470, 471 rollably slidably guidably engage the outer peripheral surfaces 40 of the side wall portions 16 of and support can body members as illustrated at 482. The side plate member 458 and rail members 470, 471 extend upwardly beyond lower vertical center line 45 approximately 16° to radial line 45 290, Figure 12. The upper portions 484 of rail members 470, 471 are located radially outwardly slightly beyond the outer peripheral surfaces of the can body members carried in the pockets 44 while extending substantially tangentially relative thereto. An 50 upper plate member 486 having a lower guide surface 488 aligned with the lower surfaces of upper rail members 461, 462 is fixedly attached to bracket 463 and extends upwardly beyond lower vertical center line 45 approximately 8° with the upper portion 490 thereof located radially inwardly slightly beyond the surfaces 106 of the pockets 44 while extending substantially tangentially to the circular path of 55 movement of the radially innermost portion of surface 106. The upper terminal portion of the plate member 486 may be curved to provide guide ramp means 492 for facilitating movement of the can body members thereon 60 65 from the transfer wheel means. Side guide plate

member 458 is substantially vertically aligned with side plate 436 of chute means 66 in axially outwardly spaced relationship to the adjacent rotating transfer wheel member 82. The upper terminal portion 494 of member 458 70 extends radially inwardly beyond and in substantial overlapping relationship with the pockets 44 to form a continuation of surface 446 of side plate member 436 for confining axial outward movement of the flange portion 20 of the can body members. Side guide plate member 459 is substantially vertically aligned with rotating plate member 94 and has an inclined upper terminal end portion 496 located radially adjacent and extending generally tangentially relative to the outer peripheral surface of plate member 94 so that the bottom rim portions 230 of the can body members are axially outwardly confined between the chute passage inlet opening 455 80 and the side plate 459 by plate member 94. 85

### Operation

In the presently preferred embodiment of 90 the aforescribed apparatus, the infeed chute means 48 is connected to a continuous supply of drawn and ironed aluminum can body members 14 which have a formed flange 20 and which are supposed to have a printed ink label 95 on the exterior peripheral surface of the side wall portion. In normal continuous operation, the solenoid operated pins 176 associated with the infeed chute 48 are withdrawn so that such can body members 14 are stacked in the chute for continuous gravity feed movement therethrough. In the event that it is desired to interrupt the continuous gravity feed of the can body members, the solenoid means 174, 175 may be actuated to extend pins 176 into 100 engagement with the can body member, one pin entering the opening 19 and extending into the interior of an axially aligned can body member in chute passage 130, the pins 176 being located on the chute in a position to be axially aligned with one of the can body members therein. 110

In continuous operation of the apparatus, the drive shaft means 36 is continuously rotated in the direction of the arrows and causes continuous rotation of the transfer wheel means 42. All loading, testing, and unloading functions are performed during one revolution of the transfer wheel means. 115

During each revolution, empty pockets 44 are rotated past the infeed means 48 whereat one can body member is gravity loaded into each pair of axially aligned pockets 44 on the transparent plate member 82, 84. As indicated in Figure 11, the initial loading occurs along radial line 169 at approximately 50° before upper vertical center line 46. As the can body member is carried further upwardly from the infeed means, the seating means 51 is effective to fully seat and maintain the can 120 125 130

body member on arcuate segment surfaces 106 of the pockets. The seating means brushes 200, 201 are initially effective at a first rotational position 502 located at about 5  $45^\circ$  before upper vertical center line 46 and substantially fully seat the can body member in the next  $5^\circ$  of generally upward rotation while being continuously effective for about  $38^\circ$  of rotation. 70

10 The unprinted can body member detector means 64 is located at a second rotational position 504 at about  $30^\circ$  beyond the first position. The detector means is a self contained conventional retroreflective photo 75

15 detector unit having a light source for applying light to the outer peripheral surface of the side wall portion 16 of the can body member and having a light detection means for generating a defective can body member control signal upon 80

20 reflection of a predetermined level of light from the outer peripheral can body member surface indicative of the absence of a printed ink label thereon. The defective can body member control signal is utilized to subsequently cause discharge of the unprinted can body member into discharge chute means 66 as hereinafter described. 85

The extendable and retractable means 52, associated with each of the aligned pairs of 90

30 pockets 44, are initially actuated at a third rotational position 506 about  $40^\circ$  before the upper vertical center line 46 and are gradually axially, slidably displaced, during approximately the next  $40^\circ$  of generally upward rotation, 95

35 from the fully retracted position to the fully extended position at about the time of reaching the upper vertical center line 46. As the means 52 are extended, abutment surface 220 on releasable holding means 54 100

40 abuttingly engages the can body member bottom wall portion 17 and axially displaces the can body member relative to the arcuate surface segments 106 of the aligned pockets 44 and relative to the seating means 51 which 105

45 maintains the can body member on and in slidable engagement with the arcuate segment surfaces 106. The axial movement of the means 52 continues until the flange portion 20 is located in abutting engagement with the 110

50 flexible sealing lip portion 342 of sealing means 56, as shown in Figure 9, at approximately the upper vertical center line 46 rotational portion 508. 115

When, or preferably slightly before, the can body member has been axially shifted to 120

55 locate the flange portion 20 in engagement with the flexible sealing lip portion 342, vacuum is applied to the releasable holding means 54 by alignment of the vacuum passage 248 in connecting plate 244 with the arcuate vacuum chamber 250 in manifold member 252. As shown in Figure 11, the vacuum passage 248 becomes aligned with vacuum chamber 250 at 125

60 rotational position 510 along radial line 280 about  $6^\circ$  before the vertical center line 46 130

so that vacuum holding of the can body member is effected before beginning the generally downward rotation after center line 46. The vacuum chamber 250 extends circumferentially about  $170^\circ$  and terminates at radial line 282 about  $16^\circ$  before the lower vertical center line 45 so that the vacuum is continuously applied to the releasable holding means for about  $170^\circ$  of rotation to effect vacuum holding of the can body members on the releasable holding means unless sooner terminated as hereinafter described.

In addition, when, or preferably slightly before, the can body member has been axially shifted to locate flange portion 20 in engagement with the flexible sealing lip portion 342, pressurized air is applied to chamber 358 and the interior of the can body member by alignment of the associated one of the air passages 362, 363 in member 86 with the associated one of the arcuate air chambers 368, 370 in manifold member 366. As shown in Figure 11, the associated air passages and air chambers become aligned along radial line 280 about  $6^\circ$  before vertical center line 46 so that pressurization of the sealing lip portion 342 and the interior of the can body member is initiated before beginning the generally downward rotational movement beyond center line 46. As shown in Figure 11, the associated air passages and air chambers remain aligned until reaching radial line 512 after about  $50^\circ$  of rotation beyond center line 46. By the time the can body member reaches the test position 514 along radial line 383, a pressurization of between 7 to 14 psi has been effected, the amount of pressurization being variably controllable so that varying test effectiveness may be obtained with higher pressurization resulting in a lower standard of acceptability and lower pressurization resulting in a higher standard of acceptability of can body members.

As shown in Figure 13, in a maximum defect detection, minimum pressure condition, the radial flange portion 342 has a convexly outward curvature such that the curved outer surface 348 thereof engages substantially only the inner annular edge 516 of the rim portion 22 of the can body member 14. Thus, substantially the entire exterior surface area of the flange portion 20 is exposed to light and all of the interior surface area remains uncovered to enable passage of light through any openings or cracks in the flange portion into chamber 358 and through transparent plate 353 to the light detection means 62. The curvature of flange portion 342 and the amount of area of engagement between exterior surface 348 and flange portion 20 can be varied by varying the air pressure in chamber 349 resulting in variations in the amount of force applied on the interior surface 350 in the general direction of the arrow 518. In a minimum defect, maximum pressure condition, the area of engagement between exterior surface 348 and flange

portion 20 is increased so that minor defects in the rim portion 22 are not detected.

Thus, as the pressurized can body member is carried into alignment with the light testing means 62, any light passing from outside the can body member through any pin holes in the wall portions or through cracks in the flange portion 20 or between the sealing lip portion 342 and the rim portion 22 will 10 be sensed by the photomultiplier tube and a defective can body member signal will be generated for purposes to be hereinafter described.

As soon as the test has been completed, the 15 application of pressurized air to chamber 358 is terminated at rotational position 520 along radial line 512, Figure 11. Then the extendable and retractable means 52 begin axial movement from the extended position to the retracted 20 position and the axial movement is completed during about the next 40° of rotation so as to be located in the fully retracted position by the time the can body member reaches rotational position 522 along horizontal center line 57. 25 During the axial movement from the extended position to the retracted position, the can body member is held on the releasable holding means 54 by vacuum application through passage 218 and chamber 234 against the can 30 body member bottom wall portion 17. Thus, the tested can body member is carried axially and slidably axially displaced relative to the aligned pockets 44 so as to be located in the retracted position illustrated in 35 Figure 8. In the retracted position, the flange portion 20 is axially spaced a substantial distance from the sealing means 56 and plate member 86 and the bottom wall portion 17 is located relatively closely axially adjacent 40 the side surface 442 of guide plate 94.

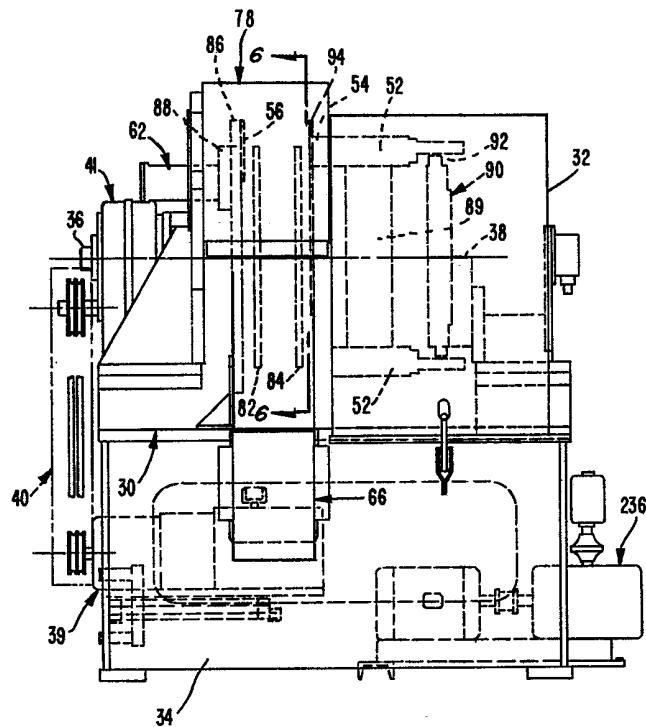
Shortly after the can body member is carried generally downwardly, beyond the horizontal center line 57, at rotational position 524 along radial line 290 10° below center line 45, the air passage 286 in member 244 is rotated into alignment with air passage 288 in manifold member 252. The flow of pressurized air in passage 288 is controlled by conventional valve means (not shown) actuatable in response 50 to a defective can body member control signal from the non-printed ink label detector means or from the light tester means 62 to permit flow of pressurized air therethrough. Thus, as air passage 286 begins to become aligned 55 with air passage 288 at rotational position 528 along radial line 529 at about 7° below horizontal center line 57 and when fully aligned at rotational position 524 about 10° below center line 57, pressurized air is delivered to 60 passage 218 in the releasable holding means 54 to dissipate the vacuum and blow-off a defective can body member.

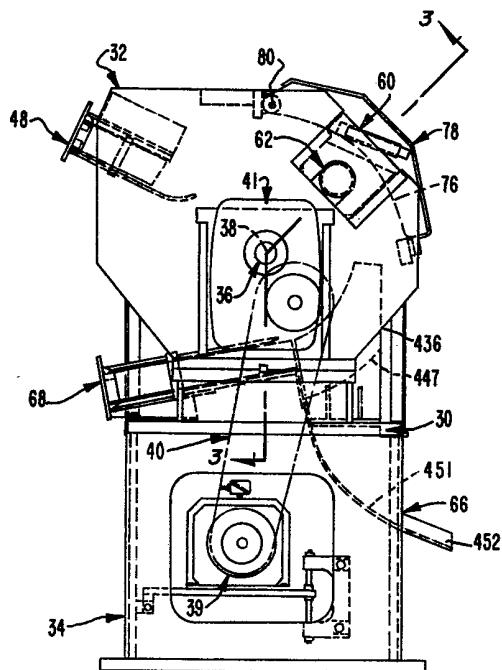
When a defective can body member is released from the releasable holding means 54 65 at about rotational position 530, the inertial

force on the can body member being carried by the rotating plate members 82, 84 cause the can body member to be separated from the pockets 44 and removed from the transfer wheel means in a generally vertical downward direction along a path of downward movement generally tangential to the circular path of movement of the pockets as indicated by arrow 532. The side surface 442 of guide plate member 94 is effective to axially confine the 70 downward movement of the defective can body member after release from the releasable holding means. In addition, the side wall 436 of the discharge chute means 66 axially opposite guide plate member 94 extends upwardly and terminates at 443 above the radial line 529, whereat application of pressurized air begins, so as to also axially confine the defective can body member during downward movement 75 in chute passage 440.

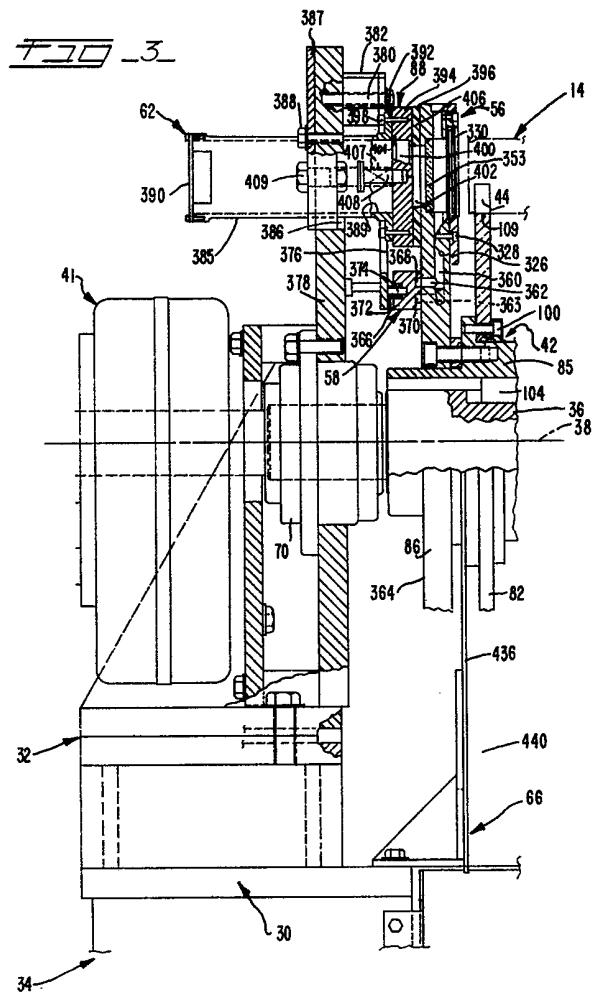
If no defective can body member signal has been generated, the vacuum is continuously applied to the releasable holding means 54 until the can body member reaches the unloading chute means 68. As shown in Figure 12, the upper inlet end of the unloading chute means extends beyond the lower vertical center line 45 about 16° so as to be substantially coterminus with vacuum chamber 250 along radial line 290. After vacuum passage 248 has been disconnected from vacuum chamber 250, air passage 286 is rotated into alignment with an air passage 534 at rotational position 536 along radial line 538. The passage 534 is continuously connected to a source of pressurized air (not shown) by suitable coupling and conduit means (not shown) similar to the means 292, 294, 296 of Figure 5. As shown in Figure 12, air passage 534 is located about 10° before lower vertical center line 45. The upper surfaces 540 of the unloading chute rail members 470 extend generally tangentially relative to the circular path of movement of the pockets 44 at the radial line 538 so that as the can body members are released from the releasable holding means 54, the inertial force of the can body members is directed substantially parallel to the upper surfaces of the unloading chute means 68, as indicated by arrow 542, to enable removal of the can body members from the pockets of the transfer wheel means by inertial force and the effect of gravity. As the released can body member is moved further toward the vertical center line 45, the upper guide plate 486 is effective to disassociate the can body member from the pockets 44 by relative movement between the can body member and surfaces 108, 114 of the plate members 82, 84. The can body members are rapidly discharged into chute passage 454 and rapidly move through the chute passage to outlet opening 456 which may be connected to a gravity type chute conveyor system (not shown). If a malfunction, such as jamming of 80 85 90 95 100 105 110 115 120 125 130

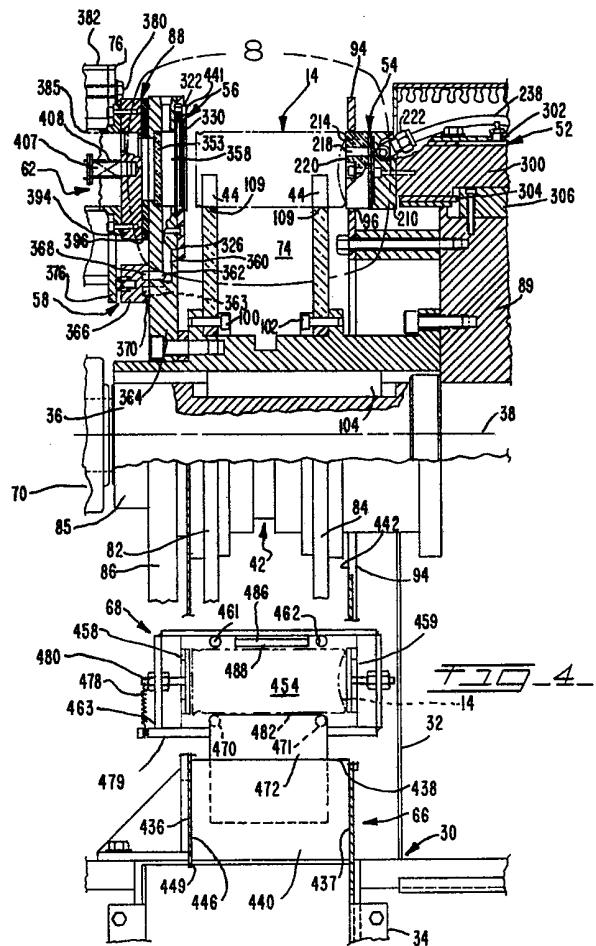
can body members, occurs in the unloading operation, creating a high force condition, the rail members 470 will be downwardly pivoted against the bias of spring 5 478 to enable removal and discharge of can body members toward the discharge passage 440 and actuate a limit switch (not shown) to terminate operation of the apparatus.	container member to be tested and through an open end portion of the container member, and between the sealing means and the annular flange portion of the container member; and	55
Thus, methods and apparatus have been 10 provided for very rapid and very effective testing of can body members during one revolution of the transfer wheel means. Some of the inventive concepts, such as the pressurization of the can body member and the flange 15 portion 342 of the sealing means 56 during testing, may be adaptable for use in testing methods involving apparatus other than disclosed hereinbefore. In addition, the methods and apparatus of the presently 20 preferred embodiment of the inventive concepts hereinbefore disclosed may be utilized in whole or in part for testing other types of container members including body members of three piece can type 25 containers and end closure members for either two piece or three piece can type containers. Furthermore, the inventive concepts may be variously otherwise modified and adapted for use in alternative embodiments thereof and for 30 use with other types of articles. Consequently, it is intended that the following claims be construed to include such alternative embodiments except insofar as limited by the prior art.	pressurization means for applying pressurized air against said sealing means to establish a predetermined amount of area engagement between said sealing means and the flange portion of the container member.	60
35 The machine for a method of continuously testing one piece can body members disclosed herein are also described and claimed in our copending British Patent Application No. 25880/77 Serial No. 1563559.	2. The invention as defined in claim 1, and wherein:	65
40 WHAT WE CLAIM IS:	said sealing means having a sealing surface engageable with said flange portion of the container member; and	65
1. Apparatus for testing wall and flange portions of a container member having an annular flange portion for defects and comprising:	said pressurization means being effective to cause said sealing means to have a convex configuration and said sealing surface to have a convex curvature for establishing a minimum area of sealing engagement between a limited portion of said sealing surface and said flange portion of the container member.	70
45 sealing means of resiliently flexible material for engaging the annular flange portion of the container member;	3. The invention as defined in claim 2, and wherein:	75
light applying means for applying light to the wall portions and annular flange portion of 50 the container member to be tested;	the convex configuration of said sealing means and the convex curvature of said sealing surface being variable in accordance with the pressure of the pressurized air.	75
light detection means for receiving light passing through the wall portions of the	4. A method of testing wall and flange portions of container members, having a flange portion, for defects, and comprising the steps of:	80
sealing the flange portion of the container member by engagement with a resilient flexible sealing means by pressurized air;	applying light to an exterior surface of the container member; and	85
detecting light passing inwardly through the wall portions of the container member from the exterior surfaces.	varying the pressure of the pressurized air to vary the area of engagement between the resilient flexible sealing means and the flange portion of the container.	90
55	5. The method as defined in claim 4, and further comprising:	95
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	52-54 High Holborn,	
	LONDON, WC1V 6RR	
	Agents for the Applicants	

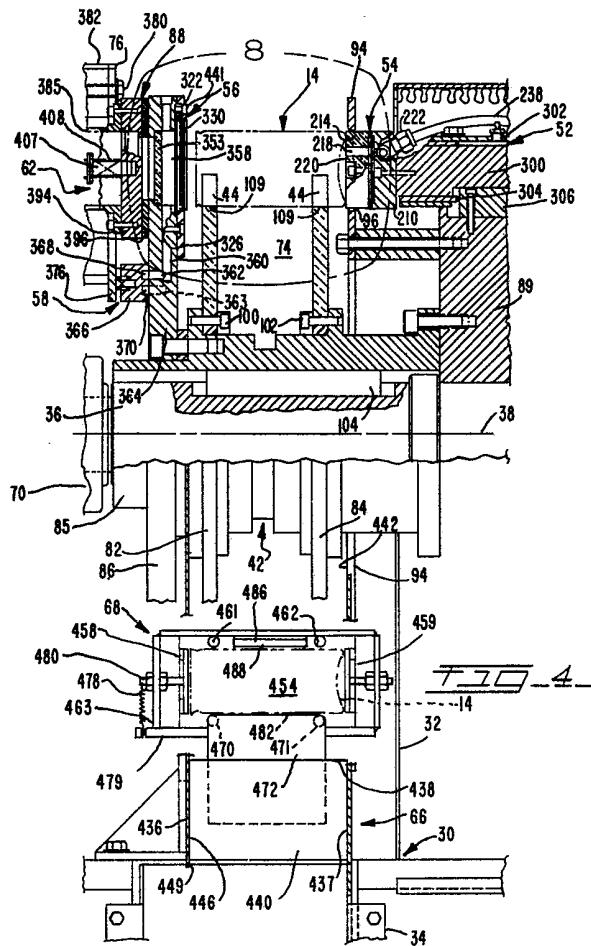




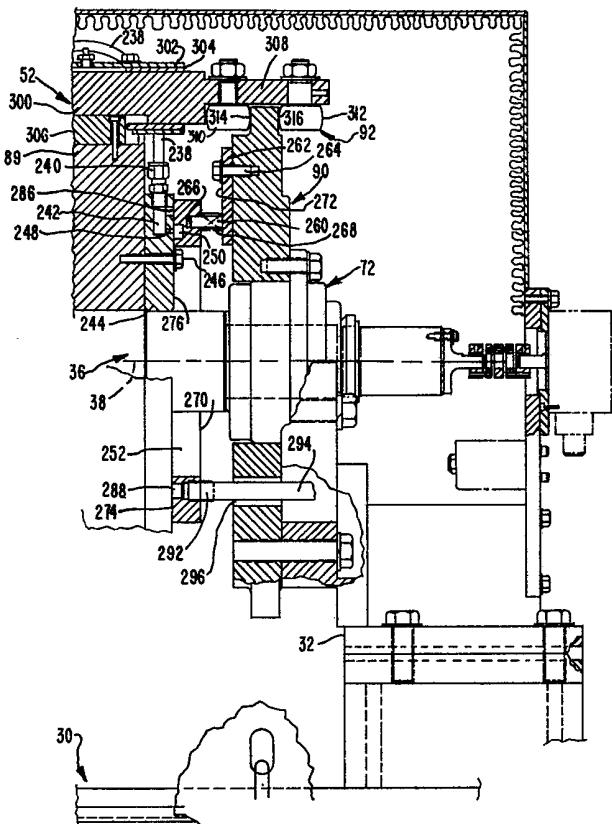
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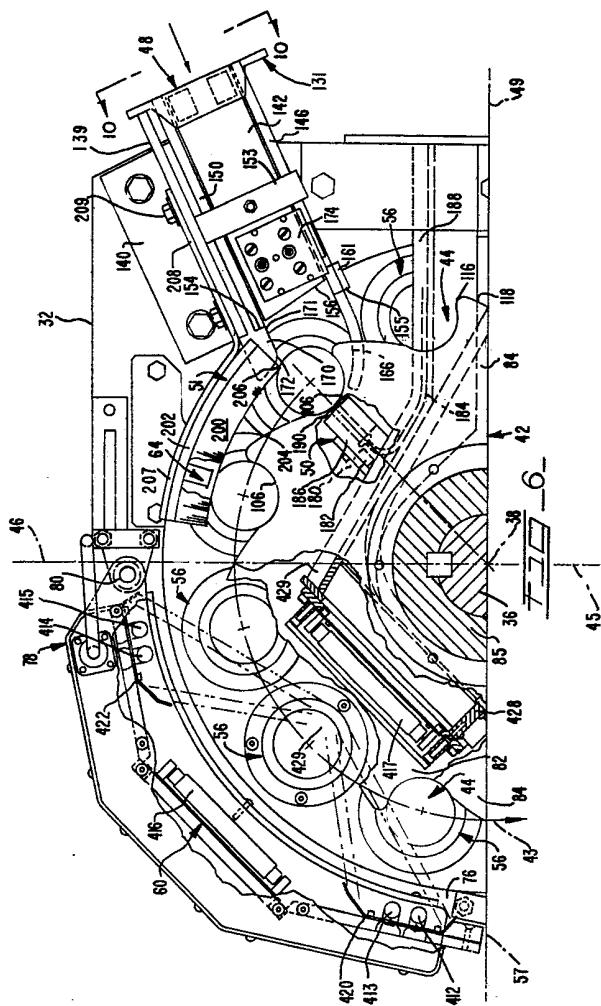


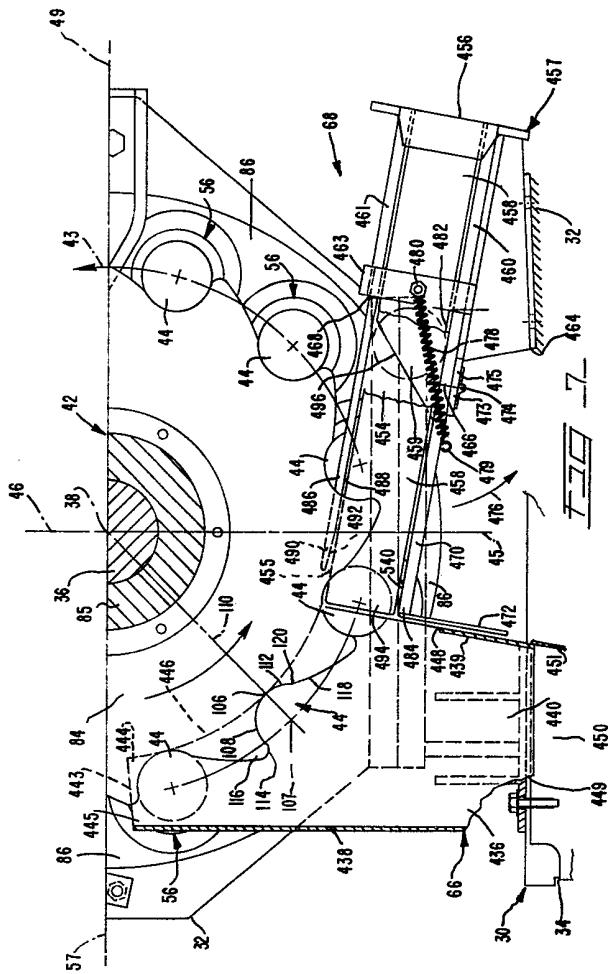


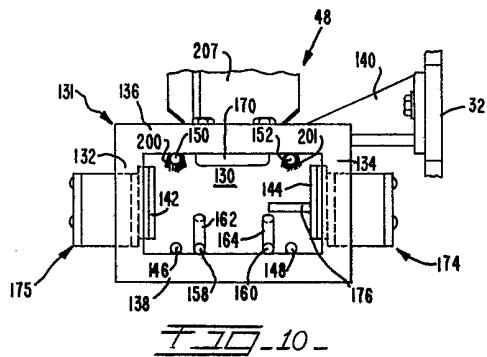
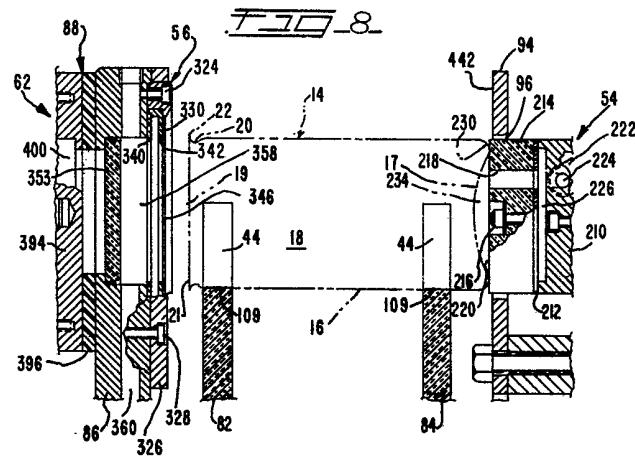


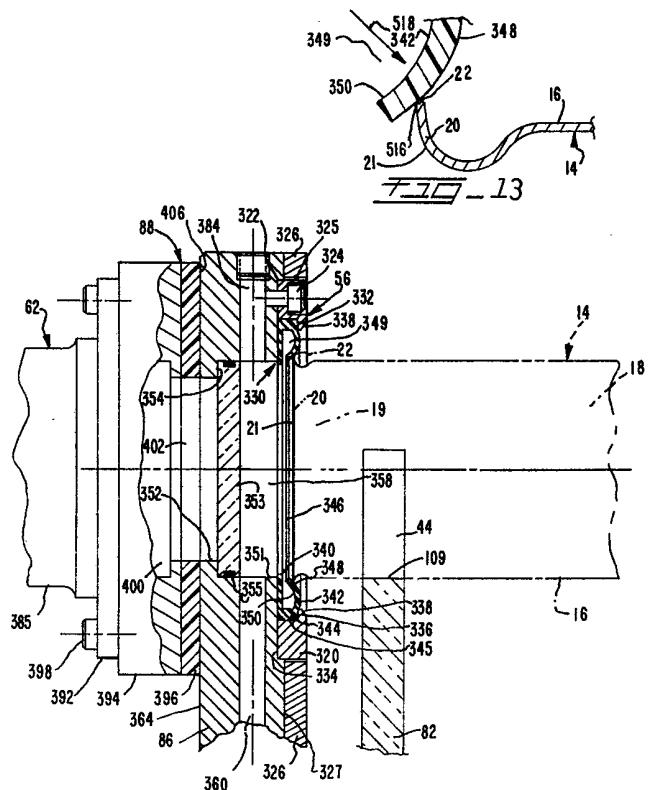
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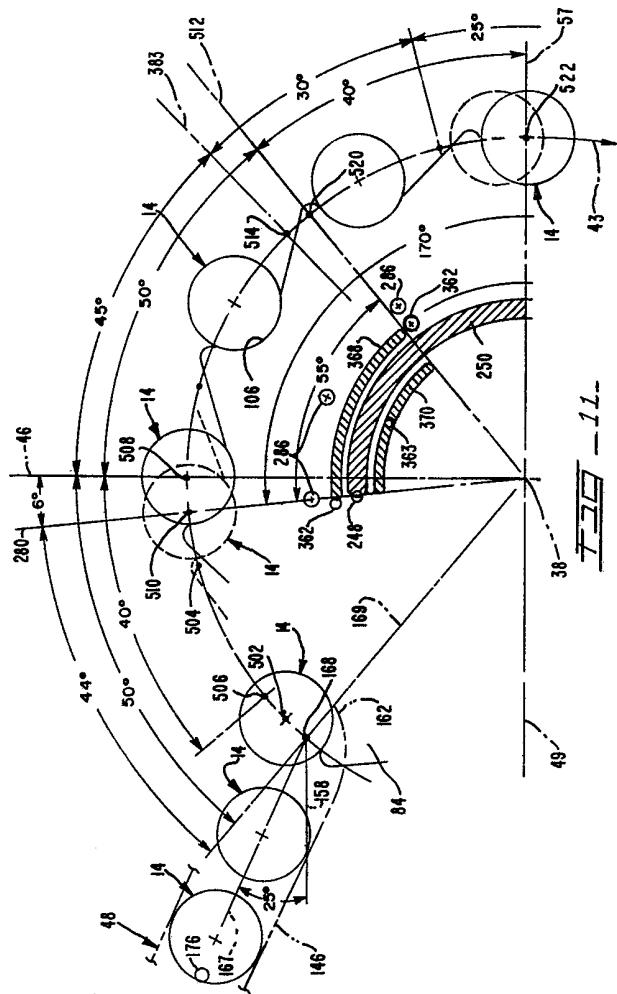


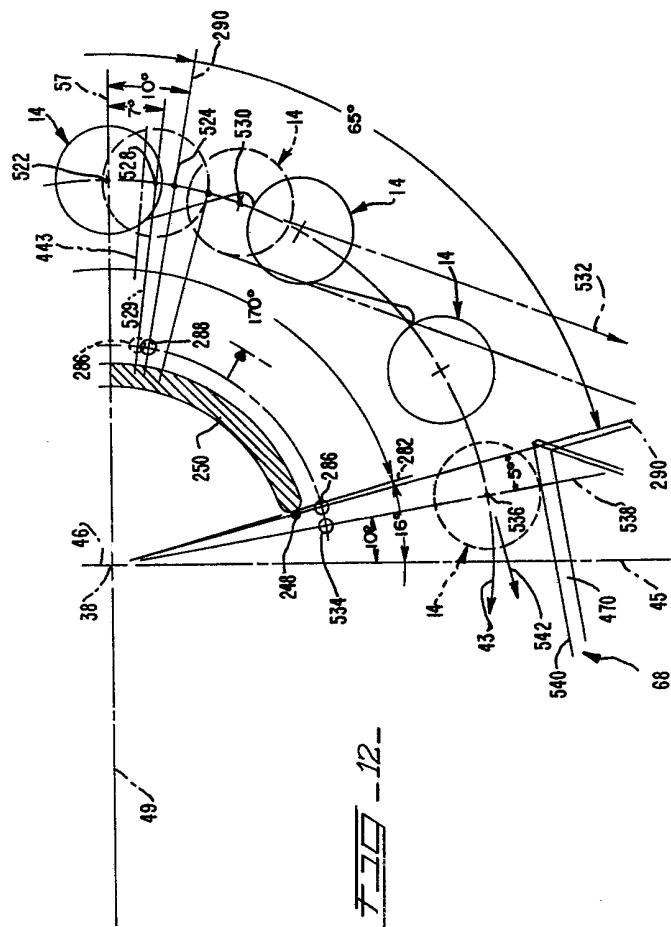


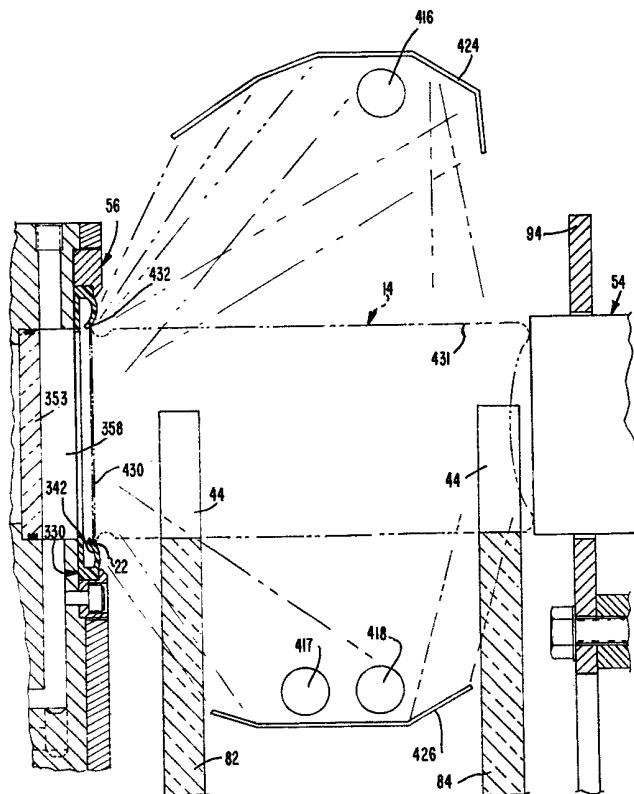




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