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(54) Title: NOZZLE VANE AND CRANK ARM ASSEMBLY AND METHOD

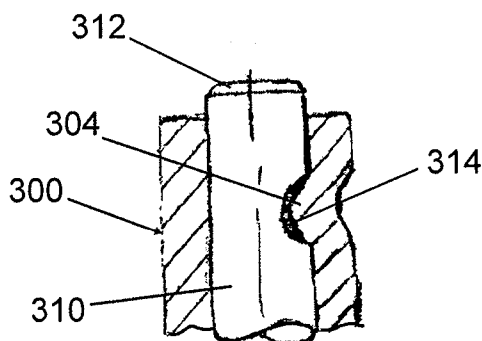


FIG. 3D

(57) Abstract: A method of assembling a crank arm (100, 200, 300) and vane assembly for a variable nozzle without requiring metallurgical bonding. In accordance with one aspect of the invention, the method comprises the steps of providing a vane assembly comprising a vane (120, 220, 320) joined to a vane shaft (110, 210, 310) that extends from the vane and terminates in a distal end (112, 212, 312), providing a recess (114, 214, 314) in an outer surface of the vane shaft at a location between the vane and the distal end, providing a crank arm (100, 200, 300) having an aperture (102, 202, 302) therein, inserting the distal end (112, 212, 312) of the vane shaft into the aperture (102, 202, 302) until the recess (114, 214, 314) in the vane shaft is inside the aperture, and causing a retaining member (104, 204, 304) associated with the crank arm (100, 200, 300) to engage the recess (114, 214, 314) in such a manner as to fasten the crank arm to the vane shaft in a substantially immovable manner.

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NOZZLE VANE AND CRANK ARM ASSEMBLY AND METHOD

BACKGROUND OF THE INVENTION

The present disclosure relates generally to a variable nozzle for a turbocharger, wherein a plurality of vanes are each respectively connected to a crank arm that can be rotated one direction or another by an actuation mechanism, such that the vanes are varied in angle in order to vary the flow through the nozzle.

5 In a conventional variable nozzle of the above-noted type, the crank arms are attached to the vanes by metallurgical bonding such as welding or brazing. This process is susceptible to variabilities, is awkward to perform because of the small sizes of the parts in a typical turbocharger nozzle, and can be expensive.

10 Additionally, in some cases the vane part to which the crank arm must be connected is non-metallic, such as ceramic. In these cases, metallurgical bonding is not possible.

BRIEF SUMMARY OF THE DISCLOSURE

This disclosure relates to an alternative method of attaching a crank arm to a vane shaft without requiring metallurgical bonding. In accordance with one aspect of the invention, a method of assembling a crank arm and vane assembly for a variable nozzle comprises the steps of providing a vane assembly comprising a vane joined to a vane shaft that extends from the vane and terminates in a distal end; providing at least one recess in an outer surface of the vane shaft at a location between the vane and the distal end; providing a crank arm having an aperture therein; inserting the distal end of the vane shaft into the aperture until the at least one recess in the vane shaft is inside the aperture; and causing at least one retaining member associated with the crank arm to engage the at least one recess in such a manner as to fasten the crank arm to the vane shaft in a substantially

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immovable manner. In some embodiments, the vane shaft has a single recess for receiving a single retaining member; in other embodiments, the vane shaft can have a plurality of recesses for respectively receiving a plurality of retaining members.

In one embodiment, a portion of the crank arm is formed as a resilient spring clip
5 that in its relaxed condition projects partially into the aperture in the crank arm. When the end of the vane shaft is inserted into the aperture, the shaft urges the spring clip radially outwardly to be flush with the inside diameter of the aperture, until the recess in the vane shaft becomes aligned with the spring clip, whereupon the spring clip springs back inwardly and engages the recess. The engagement of the spring clip in the recess
10 substantially prevents rotational movement of the shaft about its axis and translational movement parallel to the axis relative to the crank arm.

In another embodiment, the crank arm defines a hole that extends through a wall of the crank arm into the aperture. The hole extends along a direction generally perpendicular to the axis of the aperture. The vane shaft is inserted into the aperture until
15 the recess becomes aligned with the hole, and then a ball of steel or the like is inserted into the hole until it is partially engaged in the recess and partially engaged in the hole. The ball is slightly larger in diameter than the hole such that it must be pressed into the hole with an interference fit. The recess in the vane shaft also forms a very tight or interference fit with the ball. The engagement of the ball in the recess substantially prevents rotational
20 movement of the shaft about its axis and translational movement parallel to the axis relative to the crank arm.

In yet another embodiment, a portion of the wall of the crank arm that surrounds the aperture is deformed or crimped after the vane shaft is inserted into the aperture such that the portion extends into the recess with a tight fit. The engagement of the crimped
25 portion in the recess substantially prevents rotational movement of the shaft about its axis and translational movement parallel to the axis relative to the crank arm.

BRIEF DESCRIPTION OF THE SEVERAL VIEWS OF THE DRAWING(S)

Having thus described the disclosure in general terms, reference will now be made to the accompanying drawings, which are not necessarily drawn to scale, and wherein:

FIGS. 1A through 1D illustrate a crank arm and vane assembly in accordance with one embodiment of the invention;

FIGS. 2A through 2D illustrate a crank arm and vane assembly in accordance with another embodiment of the invention; and

5 FIGS. 3A through 3E illustrate a crank arm and vane assembly in accordance with yet another embodiment of the invention.

DETAILED DESCRIPTION OF THE DRAWINGS

The present invention now will be described more fully hereinafter with reference to the accompanying drawings in which some but not all embodiments of the inventions
10 are shown. Indeed, these inventions may be embodied in many different forms and should not be construed as limited to the embodiments set forth herein; rather, these embodiments are provided so that this disclosure will satisfy applicable legal requirements. Like numbers refer to like elements throughout.

A first embodiment of the invention is illustrated in FIGS. 1A through 1D. In FIG.
15 1A, a crank arm **100** is depicted in fragmentary form, it being understood that only the portion of interest is shown. The crank arm defines an aperture **102** therethrough. A portion of the wall of the crank arm surrounding the aperture forms a spring clip **104** that can be resiliently urged radially outwardly from its relaxed position. In the relaxed position, the spring clip **104** extends into the aperture **102**.

20 FIG. 1B shows the end portion of a vane shaft **110**, and FIG. 1C shows a vane assembly comprising the vane shaft **110** joined to a vane **120**. The vane shaft extends from the vane and terminates at a distal end **112**. At a location between the distal end and the vane, the outer surface of the vane shaft defines a recess **114**.

As shown in FIG. 1D, to affix the vane shaft to the crank arm, the distal end **112** of
25 the vane shaft is inserted into the aperture **102** in the crank arm until the recess **114** becomes aligned with the spring clip **104**, whereupon the spring clip springs back toward its relaxed position and engages the recess **114** in a manner substantially preventing rotational and translational movement of the vane shaft relative to the crank arm.

A second embodiment is illustrated in FIGS. 2A through 2D. In FIG. 2A, a crank arm **200** is depicted in fragmentary form, it being understood that only the portion of interest is shown. The crank arm defines an aperture **202** therethrough. A portion of the wall of the crank arm surrounding the aperture has a hole **203** formed through it. The hole
5 **203** extends along a direction generally perpendicular to the axis of the aperture, and extends into the aperture.

FIG. 2B shows the end portion of a vane shaft **210**, and FIG. 2C shows a vane assembly comprising the vane shaft **210** joined to a vane **220**. The vane shaft extends from the vane and terminates at a distal end **212**. At a location between the distal end and
10 the vane, the outer surface of the vane shaft defines a recess **214**.

As shown in FIG. 2D, to affix the vane shaft to the crank arm, the distal end **212** of the vane shaft is inserted into the aperture **202** in the crank arm until the recess **214** becomes aligned with the hole **203** in the crank arm wall. Then, a ball **204** of steel or other suitable material is pressed into the hole **203** until the ball seats into the recess **214** in
15 the vane shaft in a manner substantially preventing rotational and translational movement of the vane shaft relative to the crank arm. The ball's diameter is slightly larger than the diameter of the hole **203** such that an interference fit exists therebetween to prevent the ball from being dislodged in operation.

A third embodiment is illustrated in FIGS. 3A through 3D. In FIG. 3A, a crank
20 arm **300** is depicted in fragmentary form, it being understood that only the portion of interest is shown. The crank arm defines an aperture **302** therethrough. A portion of the wall of the crank arm surrounding the aperture has a marking or indication **303** formed thereon to signify a location at which the wall portion will subsequently be crimped as described below.

25 FIG. 3B shows the end portion of a vane shaft **310**, and FIG. 3C shows a vane assembly comprising the vane shaft **310** joined to a vane **320**. The vane shaft extends from the vane and terminates at a distal end **312**. At a location between the distal end and the vane, the outer surface of the vane shaft defines a recess **314**.

As shown in FIG. 3D, to affix the vane shaft to the crank arm, the distal end **312** of
30 the vane shaft is inserted into the aperture **302** in the crank arm until the recess **314** becomes aligned with the indication **303** on the crank arm wall. Then, a portion **304** of the

crank arm wall corresponding to the indication **303** is deformed or crimped radially inwardly until the portion seats into the recess **314** in the vane shaft in a manner substantially preventing rotational and translational movement of the vane shaft relative to the crank arm.

5 Thus, in accordance with the invention, a purely mechanical connection (as opposed to a metallurgical bond) accomplishes the attachment of the crank arm to the vane shaft. Accordingly, the process can be used with vane shafts and/or crank arms that are non-metallic such as ceramic. The process is considerably simpler, less susceptible to variabilities, and more-repeatable than welding or brazing.

10 In the various embodiments as described, the aperture in the crank arm advantageously is slightly smaller than the diameter of the vane shaft such that an interference fit exists between these parts. The recess in the vane shaft is located such that a predetermined desired spatial relationship and orientation exists between the vane shaft and the crank arm. It will be understood that the relative size and/or shape of the recess in
15 the vane shaft can be varied relative to the recess shown in the drawings. For example, the recess **114** in Figure 1 could be altered so that it essentially matches the slope and/or shape of the spring clip **104**. Additionally or alternatively, the vane shaft can have more than one recess **114, 214, 314** spaced apart circumferentially on the vane shaft, and correspondingly the crank arm can have more than one retaining member **104, 204, 304**.
20 For example, the vane shaft can have two recesses **114, 214, 314** spaced apart about 180° for receiving two retaining members **104, 204, 304** correspondingly spaced apart on the crank arm. Figure 3E shows such a variation for the embodiment of Figures 3A-D.

Many modifications and other embodiments of the inventions set forth herein will come to mind to one skilled in the art to which these inventions pertain having the benefit
25 of the teachings presented in the foregoing descriptions and the associated drawings. Although specific terms are employed herein, they are used in a generic and descriptive sense only and not for purposes of limitation.

CLAIMS

1. A method of assembling a crank arm and vane assembly for a variable nozzle, comprising the steps of:
- providing a vane assembly comprising a vane (120, 220, 320) joined to a vane shaft (110, 210, 310) that extends from the vane and terminates in a distal end (112, 212, 312);
- providing a recess (114, 214, 314) in an outer surface of the vane shaft at a location between the vane and the distal end;
- providing a crank arm (100, 200, 300) having an aperture (102, 202, 302) therein;
- 10 inserting the distal end (112, 212, 312) of the vane shaft (110, 210, 310) into the aperture (102, 202, 302) until the recess (114, 214, 314) in the vane shaft (110, 210, 310) is inside the aperture (102, 202, 302); and
- causing a retaining member (104, 204, 304) associated with the crank arm (100, 200, 300) to engage the recess (114, 214, 314) in such a manner as to fasten the crank arm
- 15 (100, 200, 300) to the vane shaft (110, 210, 310) in a substantially immovable manner.
2. The method of claim 1, wherein the retaining member comprises a resiliently deformable portion (104) of the crank arm (100) that extends into the aperture (102) in a relaxed condition, and the inserting step causes the resiliently deformable portion (104) to be urged radially outwardly until the recess (114) in the vane shaft (110) becomes aligned
- 20 with the resiliently deformable portion (104), whereupon the resiliently deformable portion (104) springs back and engages the recess (114).
3. The method of claim 1, wherein the crank arm (200) defines a hole (203) that extends through a wall of the crank arm into the aperture (202), and the causing step comprises pressing the retaining member (204) into the hole (203) until the retaining
- 25 member (204) engages the recess (214) in the vane shaft (210).
4. The method of claim 1, wherein the causing step comprises mechanically deforming a portion (304) of a wall of the crank arm (300) such that said portion (304) engages the recess (314) in the vane shaft (310).
5. A crank arm and vane assembly, comprising:

a vane assembly comprising a vane (120, 220, 320) joined to a vane shaft (110, 210, 310) that extends from the vane and terminates in a distal end (112, 212, 312);

a recess (114, 214, 314) defined in an outer surface of the vane shaft (110, 210, 310) at a location between the vane (120, 220, 320) and the distal end (112, 212, 312);

5 a crank arm (100, 200, 300) having an aperture (102, 202, 302) therein;

the distal end (112, 212, 312) of the vane shaft (110, 210, 310) extending through the aperture (102, 202, 302) and the recess (114, 214, 314) in the vane shaft (110, 210, 310) being inside the aperture (102, 202, 302); and

10 a retaining member (104, 204, 304) associated with the crank arm (100, 200, 300) engaged in the recess (114, 214, 314) in such a manner as to fasten the crank arm (100, 200, 300) to the vane shaft (110, 210, 310) in a substantially immovable manner.

6. The crank arm and vane assembly of claim 5, wherein the retaining member comprises a resiliently deformable portion (104) of the crank arm (100) that extends into the aperture (102) in a relaxed condition, the resiliently deformable portion (104) being urged radially outwardly upon insertion of the vane shaft (110) in the aperture (102) until the recess (114) in the vane shaft (110) becomes aligned with the resiliently deformable portion (104), whereupon the resiliently deformable portion (104) springs back and engages the recess (114).

7. The crank arm and vane assembly of claim 5, wherein the crank arm (200) defines a hole (203) that extends through a wall of the crank arm into the aperture (202), and the retaining member (204) is pressed into the hole (203) until the retaining member (204) engages the recess (214) in the vane shaft (210).

8. The crank arm and vane assembly of claim 5, wherein the retaining member comprises a mechanically deformed portion (304) of a wall of the crank arm (300), said portion (304) engaging the recess (314) in the vane shaft.

9. The crank arm and vane assembly of claim 5, wherein there are a plurality of recesses in the vane shaft and a corresponding plurality of retaining members.

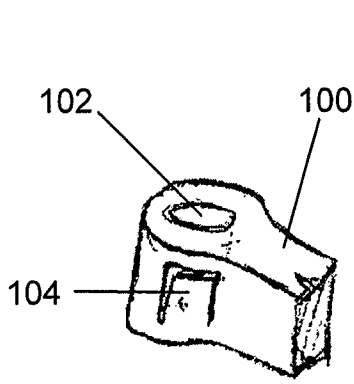


FIG. 1A

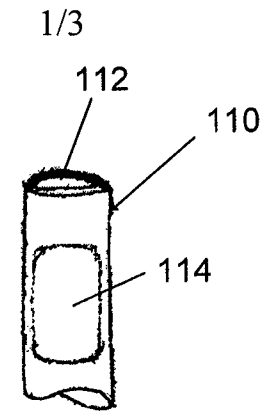


FIG. 1B

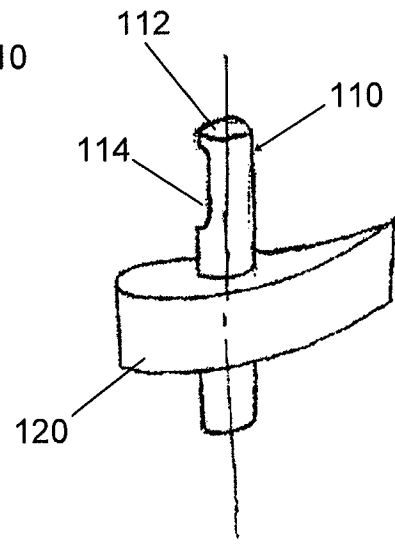


FIG. 1C

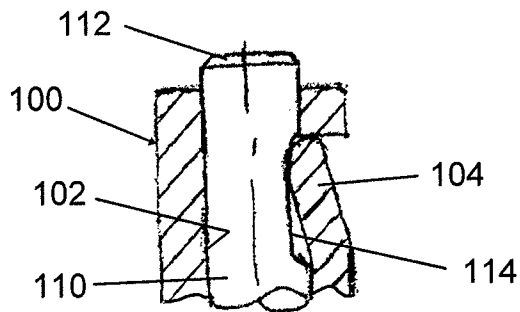


FIG. 1D

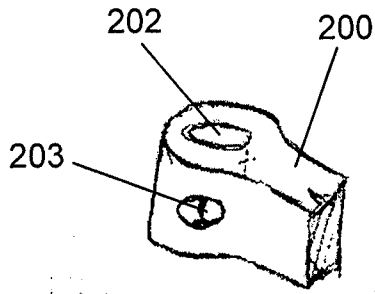


FIG. 2A

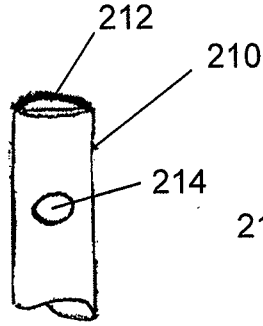


FIG. 2B

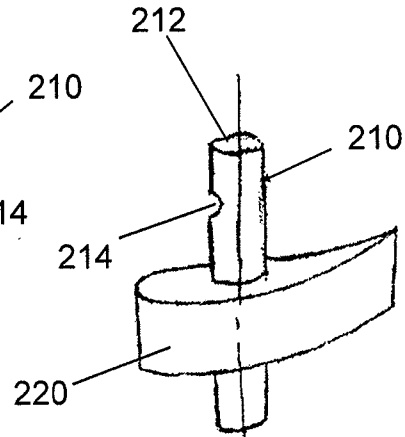


FIG. 2C

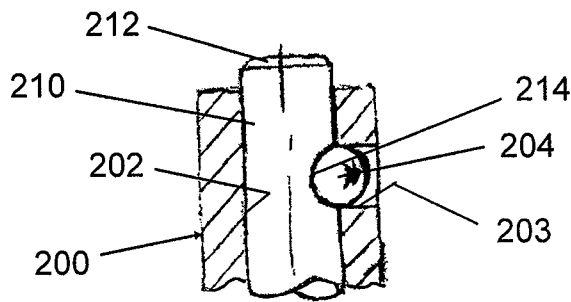


FIG. 2D

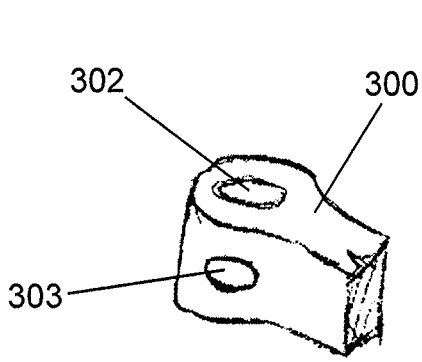


FIG. 3A

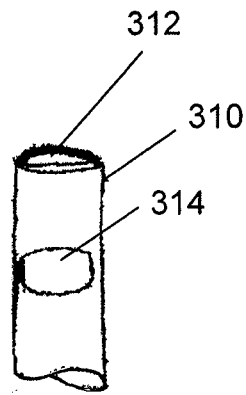


FIG. 3B

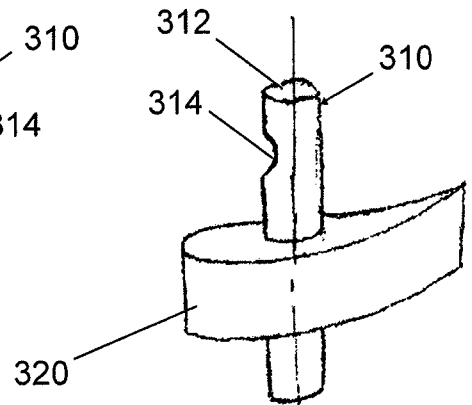


FIG. 3C

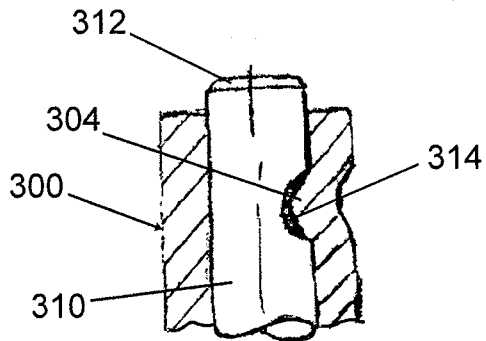


FIG. 3D

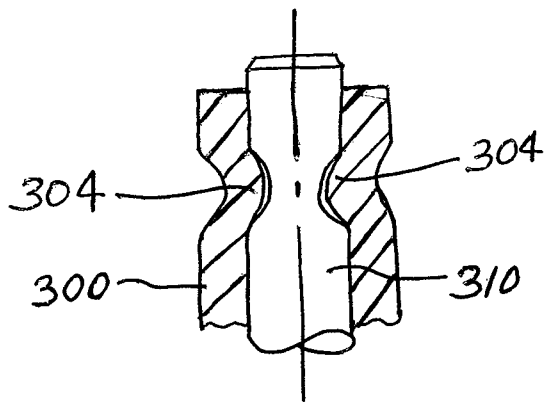


FIG. 3E

INTERNATIONAL SEARCH REPORT

International application No
PCT/US2008/086382

A. CLASSIFICATION OF SUBJECT MATTER
INV. F01D17/16

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)
F01D F02C F04D F02B

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)
EPO-Internal, PAJ, WPI Data

C. DOCUMENTS CONSIDERED TO BE RELEVANT		
Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	US 5 492 446 A (HAWKINS JAMES T [US] ET AL) 20 February 1996 (1996-02-20) column 5, lines 10-25; figure 7 -----	1,5,9
X	US 2 862 654 A (GARDINER ARTHUR W) 2 December 1958 (1958-12-02) column 3, lines 28-30; figures 3,5 -----	1,3,5,7
X	DE 21 13 194 A1 (DAIMLER BENZ AG) 28 September 1972 (1972-09-28) page 3, paragraph 2; figures 1,2 -----	1,5,9
X	US 4 299 534 A (YAMANE YOICHI ET AL) 10 November 1981 (1981-11-10) column 6, lines 10-16; figures 1,2 ----- -/--	1,5

Further documents are listed in the continuation of Box C. See patent family annex.

- * Special categories of cited documents :
- | | |
|--|--|
| <p>*A* document defining the general state of the art which is not considered to be of particular relevance</p> <p>*E* earlier document but published on or after the international filing date</p> <p>*L* document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)</p> <p>*O* document referring to an oral disclosure, use, exhibition or other means</p> <p>*P* document published prior to the international filing date but later than the priority date claimed</p> | <p>*T* later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention</p> <p>*X* document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone</p> <p>*Y* document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art.</p> <p>*G* document member of the same patent family</p> |
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Date of the actual completion of the international search 13 March 2009	Date of mailing of the international search report 27/03/2009
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Name and mailing address of the ISA/ European Patent Office, P.B. 5818 Patentlaan 2 NL - 2280 HV Rijswijk Tel. (+31-70) 340-2040. Fax: (+31-70) 340-3016	Authorized officer Teusch, Reinhold
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INTERNATIONAL SEARCH REPORT

International application No

PCT/US2008/086382

C(Continuation). DOCUMENTS CONSIDERED TO BE RELEVANT		
Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	EP 1 156 227 A (MITSUBISHI HEAVY IND LTD [JP]) 21 November 2001 (2001-11-21) figures 1a,1b,3a,3b -----	1,5,9
X	US 3 850 544 A (CIOKAJLO J) 26 November 1974 (1974-11-26) column 3, lines 6-12; figures 2,4 -----	1,5,9
X	US 5 873 700 A (ICHIKAWA HITOSHI [JP]) 23 February 1999 (1999-02-23) figures 3,4 -----	1,5

INTERNATIONAL SEARCH REPORT

Information on patent family members

International application No

PCT/US2008/086382

Patent document cited in search report	A	Publication date	Patent family member(s)	Publication date
US 5492446	A	20-02-1996	DE 19537784 A1 GB 2296049 A	20-06-1996 19-06-1996
US 2862654	A	02-12-1958	NONE	
DE 2113194	A1	28-09-1972	NONE	
US 4299534	A	10-11-1981	JP 1335632 C JP 55057667 A JP 60056276 B	11-09-1986 28-04-1980 09-12-1985
EP 1156227	A	21-11-2001	BR 0102019 A CN 1324981 A CN 1978871 A JP 2001329851 A US 2002168262 A1	26-12-2001 05-12-2001 13-06-2007 30-11-2001 14-11-2002
US 3850544	A	26-11-1974	CA 1001078 A1 DE 2444478 A1	07-12-1976 07-05-1975
US 5873700	A	23-02-1999	CN 1163348 A JP 9203371 A	29-10-1997 05-08-1997