

(19) **DANMARK**

(10) **DK/EP 3655701 T3**



(12) **Oversættelse af
europæisk patentskrift**

Patent- og
Varemærkestyrelsen

-
- (51) Int.Cl.: **F 23 D 14/24 (2006.01)**
- (45) Oversættelsen bekendtgjort den: **2024-11-18**
- (80) Dato for Den Europæiske Patentmyndigheds bekendtgørelse om meddelelse af patentet: **2024-09-18**
- (86) Europæisk ansøgning nr.: **18743477.4**
- (86) Europæisk indleveringsdag: **2018-07-19**
- (87) Den europæiske ansøgnings publiceringsdag: **2020-05-27**
- (86) International ansøgning nr.: **EP2018069612**
- (87) Internationalt publikationsnr.: **WO2019016307**
- (30) Prioritet: **2017-07-21 DE 102017116529**
- (84) Designerede stater: **AL AT BE BG CH CY CZ DE DK EE ES FI FR GB GR HR HU IE IS IT LI LT LU LV MC MK MT NL NO PL PT RO RS SE SI SK SM TR**
- (73) Patenthaver: **Kueppers Solutions GmbH, Robert-Schuman-Str. 6, 44263 Dortmund, Tyskland**
- (72) Opfinder: **TE KAAT, Jens, Stabelpfad 16, 44225 Dortmund, Tyskland**
- (74) Fuldmægtig i Danmark: **Plougmann Vingtoft A/S, Strandvejen 70, 2900 Hellerup, Danmark**
- (54) Benævnelse: **BRÆNDER**
- (56) Fremdragne publikationer:
CN-A- 106 907 710
DE-T2- 69 818 909
DE-U1- 9 208 993
US-A- 5 408 830
US-B1- 6 652 268

Description

The invention relates to a burner having a housing on which a burner pipe is disposed, wherein the burner pipe has an opening at the end turned away from the housing, wherein a mixing element is provided in the burner pipe and the space between this mixing element and the opening forms a combustion chamber, wherein the housing has at least two ducts that are separated from one another and open in the mixing element, wherein gases flow through the ducts and the mixing element and a mixing of these gases first takes place in the combustion chamber.

Such burners, especially gas burners for industrial use are devices that transform chemical energy to thermal energy in a combustion process. In this combustion process, at least one oxidant, preferably air or oxygen, is consumed together with a fuel in a continuous heat-releasing reaction in the combustion chamber. The heated exhaust gases are discharged via the opening into the open environment and may be used, for example, in drying systems, thermal post-combustion facilities, circulation systems, curing ovens or other process-engineering systems. All commercial, purified gases such as city gas, grid gas, natural gas and liquefied gas as well as their mixtures are examples of fuels that may be fired together with air.

The burners generally known from the state of the art can be divided into two categories, wherein the first category includes burners in which the oxidant is injected as a mixture together with the fuel into the combustion chamber. Such burners are also known as pre-mix burners. In contrast to the burners of the first category, the burners included in the second category have separate supply lines for the oxidant and fuel to the combustion chamber, and so mixing of the two substances first takes place in the combustion chamber.

The burners of the first category indeed have the advantage that the mixture of oxidant and fuel is available in a homogeneous condition in the combustion chamber, but they also have the disadvantage that an absolute safety against flashbacks is not possible. In burners of the second category, flashbacks are precluded by the separate feed of the oxidant and of the fuel, but the homogeneity of the mixture in the combustion chamber is not optimal compared with the homogeneity achieved by the mixture of a burner of the first category.

From US 2016/0377293 A1, from US 2016/0223201 A1 as well as from GB 2 536 965 A, additive fabrication processes for the manufacture of structural parts for burners

are known, since this is appropriate precisely for complexly configured structural parts. A burner of this type is known from US 5 408 830.

The invention relates to a burner of the second category and, in view of the above-described range of constraints between safety and optimum combustion due to a high
5 degree of homogeneity of the mixture, has set itself the task of specifying a burner of the second category, which achieves an improved homogeneity of the mixture in the combustion chamber, in order to optimize the combustion process.

This object is achieved by a burner of the type designated in the introduction according to the invention with the features of claim 1.

10 Thus, a burner is made available in which the mixing element is manufactured with an additive fabrication process. The additive fabrication processes also include, among others, powder-bed processes, especially the selective laser melting as well as the selective laser sintering. Due to the fabrication of the mixing element with an additive
15 fabrication process, a new type of feed of the oxidant and of the fuel into the combustion chamber can be achieved, which is optimizable to the effect that a homogeneous mixing is achievable over a broad regulation range of the ratio of oxidant to fuel and hereby a complete combustion over the entire power range of the burner is assured.

This mixing element has at least two separate intermediate ducts, which branch in flow
20 direction in the direction of the combustion chamber. Intermediate ducts that branch and then open out are indeed known from US 2016/0223201 A1 , but that publication relates not to any gas burner but instead to a fuel injector for a gas-turbine engine. It involves a fuel injector, which is to be compared with gas burners of the first category, in which the oxidant is injected together with the fuel as a mixture into the combustion
25 chamber (a so-called premix burner).

The invention, however, relates to a burner of the second category, in which the separate supply of the oxidant as well as of the fuel takes place via two ducts up to the combustion chamber. US 2016/0223201 A1 , however, has only one main feed duct, as is clearly apparent from Fig. 2. On this there is disposed a node, from which
30 two subordinate lines (not separated from one another) branch off inside the turbine.

A problem that is not to be underestimated during operation of a burner, is the formation of harmful nitrogen oxides, which may form during the combustion process. In this context, the configuration of the burner, especially of the combustion chamber, and the feed of the oxidant and of the fuel, have a considerable influence on the

formation of the nitrogen oxide during the combustion process. An additively fabricated mixing element may be optimized to the effect that the formation of such nitrogen oxides is either suppressed completely or reduced to a minimum. In this connection, it is conceivable that the burner provides an exhaust-gas return, in which the already
5 burned exhaust gases having a low oxygen content and a relatively high carbon dioxide content are introduced into the combustion chamber. The injection of such inert exhaust gases leads to a distinct reduction of the formation of the nitrogen oxides. Furthermore, a configuration of the burner and/or mixing element is also conceivable to the effect that the exhaust-gas return provides a preheating of the oxidant and/or
10 fuel being fed into the combustion chamber.

Further configurations of the invention will become apparent from the dependent claims.

In this connection, it is expedient for the mixing element to be constructed in one piece. A one-piece mixing element has the advantage that the feed of the oxidant and of the
15 fuel into the combustion chamber can be optimized with a compact dimension of the mixing element. Moreover, the feed of the oxidant and of the fuel can be provided in a way that is not realizable with a multi-piece mixing element. At the same time, the invention provides that the mixing element has at least two separate intermediate ducts, which branch in flow direction in the direction of the combustion chamber. The
20 branching of the intermediate ducts ensures that the oxidant and/or the fuel are injected not only locally at one point into the combustion chamber but instead at a multiplicity of possible positions, so that a largely homogeneous mixing of the oxidant and of the fuel already takes place in the initial region of the combustion chamber. The
25 said initial region of the combustion chamber is to be regarded as the region that has a shorter distance to the mixing element than to the opening of the burner. The homogeneous mixing of the oxidant and of the fuel not only suppresses the formation of nitrogen oxides but also increases the efficiency of the burner and thus reduces the fuel costs.

Furthermore, it is preferably provided that three to five intermediate ducts are
30 provided. This choice represents an optimum between the complexity of the construction of the mixing element and its fabrication-related requirements as well as the advantages to be achieved in terms of an optimum combustion.

In a further configuration, it is provided that the intermediate ducts open in a multiplicity of outlet openings, wherein, in at least one intermediate duct, the outlet cross-section

is reduced compared with the duct cross-section. The reduction of the outlet cross-section offers the advantage that, on the side of the mixing element turned toward the combustion chamber, a multiplicity of outlet openings can be provided that permit an optimum mixing of the oxidant and of the fuel. On the other hand, it is further possible to feed a high volume flow of gases to the mixing elements, since the ducts leading to the mixing elements have a relatively large diameter.

Furthermore, it is provided that each individual outlet opening is equipped with suitable mixing, swirling and/or atomizing elements for the gases at the outlet from the mixing element. The gases are turbulently swirled by these elements at the outlet, so that the oxidant can be mixed particularly homogeneously with the fuel. It is advantageous in this context that the homogeneity of the mixing of the oxidant and of the fuel is already available at the inlet into the initial region of the combustion chamber.

A further configuration provides that swirl elements, baffle elements, flow-separating edges, stagnation edges, grooves, mixing nozzles, mixing valves and/or outlet elbows are provided on the outlet openings. By this, it is to be understood not that all outlet openings have the same elements but instead that, depending on position of the outlet opening, one of the aforesaid elements that leads on the whole to an improved homogeneity of the mixing is provided. This permits a diverse combination of the most diverse elements, so that the mixing element can be adapted to the requirements of particular burner classes.

A special configuration of the invention provides that at least one of the intermediate ducts has helical grooves. In this case, it is possible, for example, that the respective groove has a rectangular cross-section or is even diamond shaped. By the introduction of helical grooves into the intermediate ducts, the inlet velocity of the respective gases into the combustion chamber can be controlled, in order to achieve mixing of the oxidant and of the fuel with a high degree of homogeneity.

A further particular configuration of the invention provides that the outlet openings are disposed in such a way that the gases achieve a homogeneous mixing in the combustion chamber. Thereby the efficiency of the burner is increased and simultaneously the formation of harmful nitrogen oxides is suppressed. The feed of the oxidant and of the fuel can take place via several ducts, wherein, depending on the power range, individual feeds, for example, are blocked or opened in the region of the housing of the burner. Hereby, a complete combustion can be achieved over the entire power range of a burner.

Moreover, it is preferably provided that a UV or IR flame sensor is provided on the outlet side of the mixing element. Hereby, the combustion process can be monitored continuously, in order to adapt the feed of oxidant and/or fuel in such a way if necessary that the combustion process proceeds according to the desired criteria. For this purpose, the flame sensor may be combined with a suitable control or regulating unit using information technology, wherein this control or regulating unit may be provided, for example, in the housing of the burner.

According to the invention, it is provided that exhaust gas is able to flow back from the combustion chamber in the direction of the housing through one or more of the intermediate ducts. In particular, via the wall of the intermediate ducts, the returning exhaust gas is able to preheat the gases flowing inward in the direction of the combustion chamber. Since the cold combustion air flows around the ducts conveying the hot exhaust gas, the combustion air is preheated in the process. This increases the efficiency of the burner in the sense of a recuperative burner, wherein the special advantages of the heat-exchanger manufactured by 3D printing are fully manifested. Hereby, a very compact heat exchanger with good efficiency is obtained inside the mixing unit.

Further features, details and advantages of the invention will become apparent on the basis of the following description hereinafter as well as on the basis of the drawings. Items or elements corresponding to one another are denoted by the same reference symbol in all figures, wherein

- Fig. 1 shows a schematic diagram of a burner according to the invention,
- Fig. 2 shows a perspective view of a mixing element according to the invention, which is designed as a two-duct system,
- 25 Fig. 3 shows an imaginary exploded diagram of the mixing element from Fig. 2,
- Fig. 4 shows an inlet-side view of the mixing element from Fig. 2,
- Fig. 5 shows an outlet-side view of the mixing element from Fig. 2,
- Fig. 6 shows a sectional diagram of the mixing element from Fig. 2 along the line A-A,
- 30 Fig. 7 shows a perspective view of a mixing element according to the invention, which is designed as a three-duct system,
- Fig. 8 shows an imaginary exploded diagram of the mixing element from Fig. 7,
- Fig. 9 shows an inlet-side view of the mixing element from Fig. 7,
- Fig. 10 shows an outlet-side view of the mixing element from Fig. 7,

Fig. 11 shows a perspective view of a mixing element according to the invention, which is designed as a four-duct system,

Fig. 12 shows an imaginary exploded diagram of the mixing element from Fig. 11,

Fig. 13 shows an inlet-side view of the mixing element from Fig. 11,

5 Fig. 14 shows an outlet-side view of the mixing element from Fig. 11,

Fig. 15 shows a perspective view of a mixing element according to the invention,

Fig. 16 shows a variant of Fig. 15 with indicated gas arrows, and

Fig. 17 shows an inlet-side view of the mixing element having a recirculator.

In Fig. 1, a schematic diagram is shown of a burner according to the invention, which
10 has a housing 2, on which a burner pipe 3 is disposed, wherein the burner pipe 3 has an opening 4 at the end turned away from the housing 2. A mixing element 5 is provided in the burner pipe 3, and the space between this mixing element 5 and the opening 4 forms a combustion chamber 6. In this exemplary embodiment, a first duct 8 extends at least partly through the housing 2 and the burner pipe 3. The spatial
15 volume or the intermediate chamber 7 between the burner pipe 3 and the first duct 8 forms a further duct.

Thus the housing 2 has, separated from one another, at least two ducts 8, which open in the mixing element 5, wherein gases propagate or flow through the ducts 8 and the mixing element 5, and a mixing of these gases first takes place in the combustion
20 chamber 6. For reasons of clarity, the connection between the housing 2 and the duct formed by the intermediate chamber 7 is not illustrated. In the combustion chamber 6, a UV or IR flame sensor is provided on the outlet side of the mixing element 5, in order to monitor the combustion process continuously. What is not illustrated is an igniter, which ignites the gas mixture in the fuel chamber 6.

25 The mixing element 5, which is designed as a two-duct system, is shown in Fig. 2 in a perspective view, in Fig. 3 in an imaginary exploded diagram, in Fig. 4 on the inlet side and in Fig. 5 on the outlet side, and also in Fig. 6 as a sectional diagram. The imaginary exploded diagram in Fig. 3 provides merely an overview, and separation of the various components of the mixing element 5 is actually not possible, because it would lead to
30 a destruction of the mixing element 5, since the mixing element 5 is formed in one piece by means of an additive fabrication process. The inlet side is to be understood as that side of the mixing element 5 that in the installed condition in the burner 1 adjoins the intermediate chamber 7. The outlet side is to be understood as that side

of the mixing element 5 that in the installed condition in the burner 1 adjoins the combustion chamber 6.

The mixing element 5 preferably has a tubular outer wall 14, which is equipped with radially inwardly projecting flow-resistance elements 16, wherein the flow-resistance elements 16 are preferably aligned transversely relative to the longitudinal axis of symmetry or axis of rotation of the outer wall 14. The flow-resistance elements 16 may be separated in radial direction by tube portions, which have a smaller diameter and a shorter longitudinal extent than the outer wall 14. Hereby it is possible that the flow-resistance elements 16, depending on their radial position, are disposed at various angles transverse relative to the longitudinal axis of symmetry of the outer wall 14. Preferably, the flow-resistance elements 16 may be designed in the manner of paddle wheels.

The inner or first duct 8, through which preferably the gaseous fuel is conveyed, is branched into a multiplicity of intermediate ducts 9, which pass through the flow-resistance elements 16 and their associated tube portions. The intermediate ducts 9 have outlet openings 10, which open in the combustion chamber 6. For the sake of clarity, possible swirl elements, baffle elements, flow-separating edges, stagnation edges, grooves, mixing nozzles, mixing valves and/or outlet elbows at the outlet openings 10 are not illustrated. The second duct 8 or the intermediate chamber 7 for the oxidant is formed by the space bounded by the outer wall 14 and the duct 8 as well as its intermediate ducts 9. What is advantageous in this exemplary embodiment is that the intermediate ducts 9 themselves represent additional flow-resistance elements. Thus, the intermediate ducts 9 have not only the function of transporting the fuel into the combustion chamber but also that of contributing to swirling of the oxidant. In this embodiment, the duct cross-section 12 of the first duct 8 is made larger than the cross-section of the outlet openings 10, in order to achieve a uniform distribution of the fuel.

A further mixing element 5, which is designed as a three-duct system, is shown in Fig. 7 in a perspective view, in Fig. 8 in an imaginary exploded diagram, in Fig. 9 on the inlet side and in Fig. 10 on the outlet side. The imaginary exploded diagram in Fig. 8 provides merely an overview, and separation of the various components of the mixing element 5 is actually not possible.

In contrast to the two-duct system, the three-duct system has an additional duct 8, which is disposed here, for example, in the first duct 8. This additional duct 8 in turn

has its own intermediate ducts 9, which pass through the flow-resistance elements 16 and their associated tube portions. This additional duct 8 may be used, for example, to inject additional fuel into the combustion chamber 6 and thereby to optimize the combustion process.

5 A special mixing element 5, which is designed as a four-duct system, is shown in Fig. 11 in a perspective view, in Fig. 12 in an imaginary exploded diagram, in Fig. 13 on the inlet side and in Fig. 14 on the outlet side. The imaginary exploded diagram in Fig. 12 provides merely an overview, and separation of the various components of the mixing element 5 is actually not possible.

10 In contrast to the three-duct system, the four-duct system has a tubular intermediate wall 15, which has a smaller diameter than the outer wall 14 but a substantially identical longitudinal extent.

As follows in particular from Figs. 15 and 16, the combustion air is able to flow through the inner ducts 8 into the combustion chamber 6 and to flow out again through one of the outer ducts 8. In order to change the flow direction and to feed the inwardly flowing exhaust gas to the combustion air again, a recirculator 17 is provided (Fig. 17).

15 Naturally, the invention is not limited to the illustrated exemplary embodiments. Further exemplary embodiments are possible without departing from the basic ideas. It is self-evident that the feed of the oxidant and of the fuel for a burner having a mixing element
 20 designed as a three-duct system or four-duct system must be appropriately adapted. Furthermore, the number of ducts is not limited to the exemplary embodiments illustrated above by way of example. Depending on requirements and field of use of the burner, even more ducts may be practical.

List of reference numerals:

25	1	Burner
	2	Housing
	3	Burner pipe
	4	Opening
	5	Mixing element
30	6	Combustion chamber
	7	Intermediate chamber
	8	Duct
	9	Intermediate duct
	10	Outlet opening

- 11 Outlet cross-section
- 12 Duct cross-section
- 13 UV or IR flame sensor
- 14 Outer wall
- 5 15 Intermediate wall
- 16 Flow-resistance element
- 17 Recirculator

Patentkrav

1. Brænder (1) med et hus (2), på hvilket et brænderør (3) er anbragt, hvilket
5 brænderør (3) har en åbning (4) i den bort fra huset (2) vendende ende, hvor der i
brænderøret (3) er tilvejebragt et blandingselement (5), og rummet mellem dette
blandingselement (5) og åbningen (4) danner et brændkammer (6), idet huset (2)
har mindst to fra hinanden adskilte kanaler (8), der udmunder i
blandingselementet (5), hvorved gasser strømmer gennem kanalerne (8) og
10 blandingselementet (5), og en sammenblanding af disse gasser først finder sted i
brændkammeret (6), idet blandingselementet (5) har mindst to adskilte
melleumkanaler (9), der forgrener sig i gennemstrømningsretningen i
brændkammerets (6) retning,

kendetegnet ved, at blandingselementet (5) er fremstillet ved en additiv
15 fremstillingsfremgangsmåde, og

at forbrændingsgas kan strømme tilbage fra brændkammeret (6) i retning mod
huset (2) gennem en eller flere af melleumkanalerne (9).

2. Brænder (1) ifølge krav 1,
20 **kendetegnet ved,**
at blandingselementet (5) er udformet i et stykke.

3. Brænder (1) ifølge krav 1 eller 2,
kendetegnet ved,
25 **at** der er tilvejebragt tre til fem melleumkanaler (9).

4. Brænder (1) ifølge et eller flere af kravene 1 til 3,
kendetegnet ved,
at melleumkanalerne udmunder i et stort antal udgangsåbninger (10), hvor
30 udgangstværsnittet (11) er formindsket i forhold til kanaltværsnittet (12) i mindst

en mellemkanal.

5. Brænder (1) ifølge krav 4,

kendetegnet ved,

- 5 **at** hver enkelt udgangsåbning (10) er udstyret med sammenblandings-, ophvirvlings- og/eller forstøvningselementer for gasserne ved udgangen fra blandingselementet (5).

6. Brænder (1) ifølge krav 4 eller 5,

10 **kendetegnet ved,**

at der på udgangsåbningerne (10) er tilvejebragt snoningselementer, prelementer, strømningsafbrydelseskanter, stuvekanter, noter, blandedyser, blandeventiler og/eller udgangsmanifolder.

15 **7.** Brænder (1) ifølge et eller flere af kravene 1 til 6,

kendetegnet ved,

at mindst en af mellemkanalerne (9) har skruelinjeformige noter.

8. Brænder (1) ifølge et eller flere af kravene 4 til 6,

20 **kendetegnet ved,**

at udgangsåbningerne (10) er anbragt således, at gasserne ved det store antal udgangsåbninger (10) med formindskelse af udgangstværsnittet (11) i forhold til kanalværsnittet (12) med særskilt styrbare mellemkanaler (9) opnår en homogen blanding i brændkammeret (6).

25

9. Brænder (1) ifølge et eller flere af kravene 1 til 8,

kendetegnet ved,

at der på blandingselementets (5) udgangsside er tilvejebragt en UV- eller IR-flammesensor (13).

30

10. Brænder (1) ifølge et eller flere af kravene 1 til 9,

kendetegnet ved,

3

at den tilbagestrømmende forbrændingsgas afgiver varme til de i
brændkammerets (6) retning indstrømmende gasser gennem mellemkanalernes
(9) væg.

5

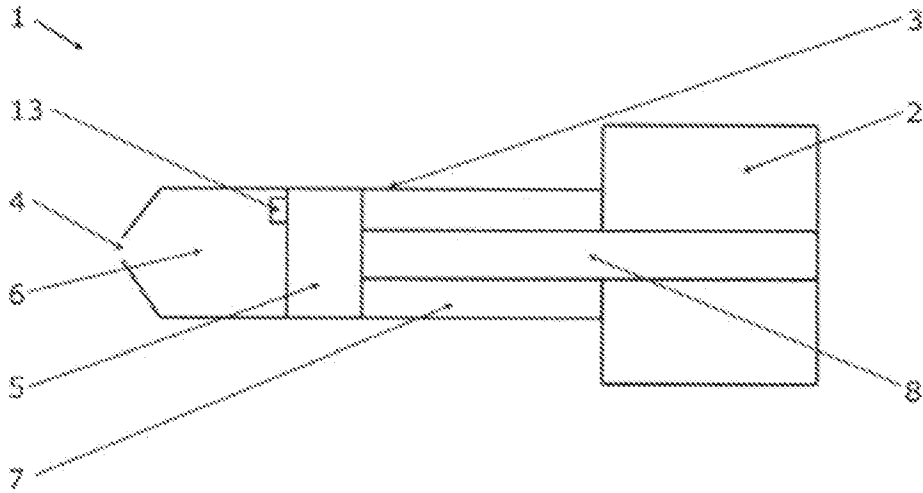


Fig. 1

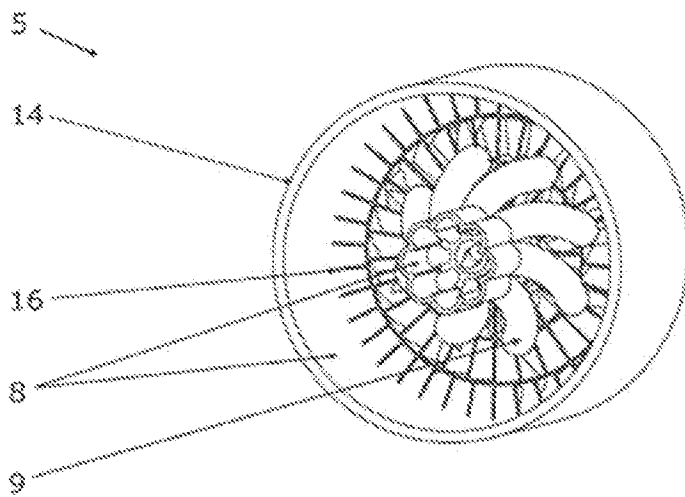


Fig. 2

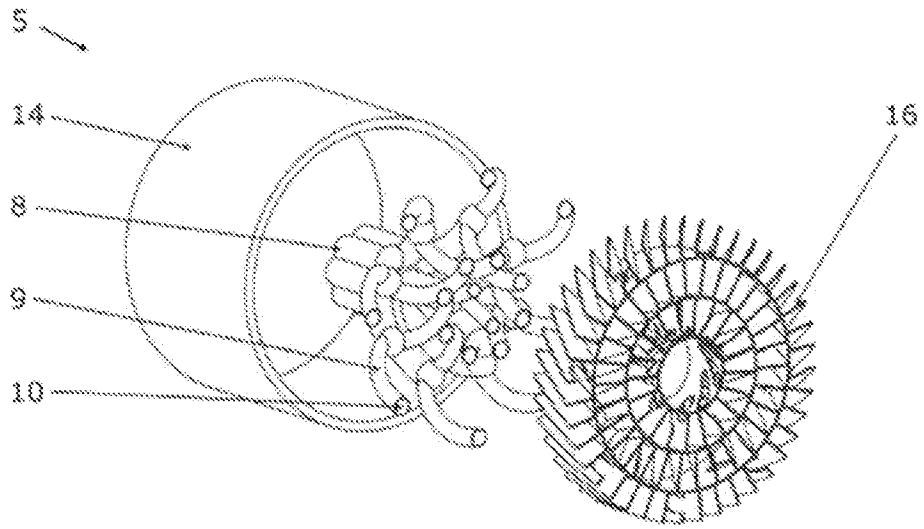


Fig. 3

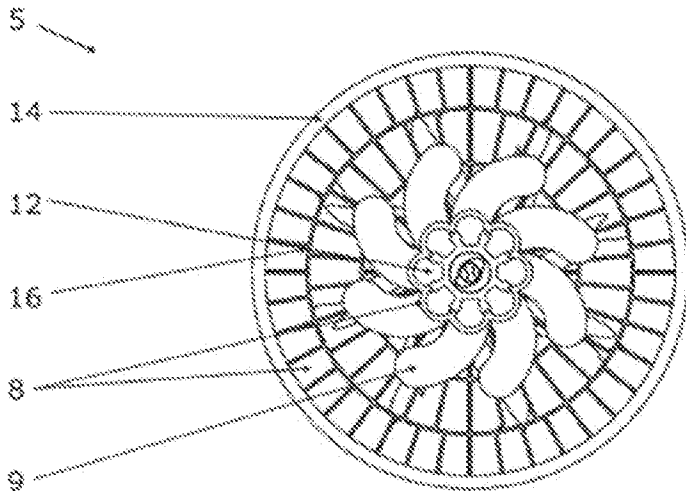


Fig. 4

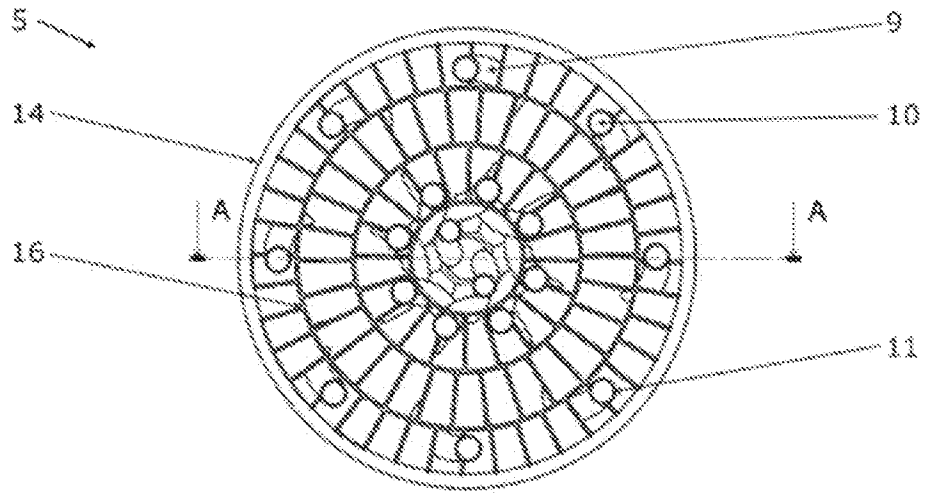


Fig. 5

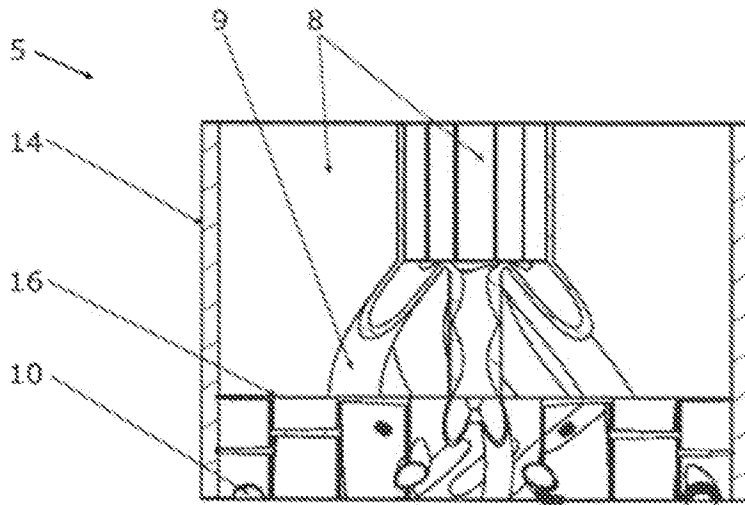


Fig. 6

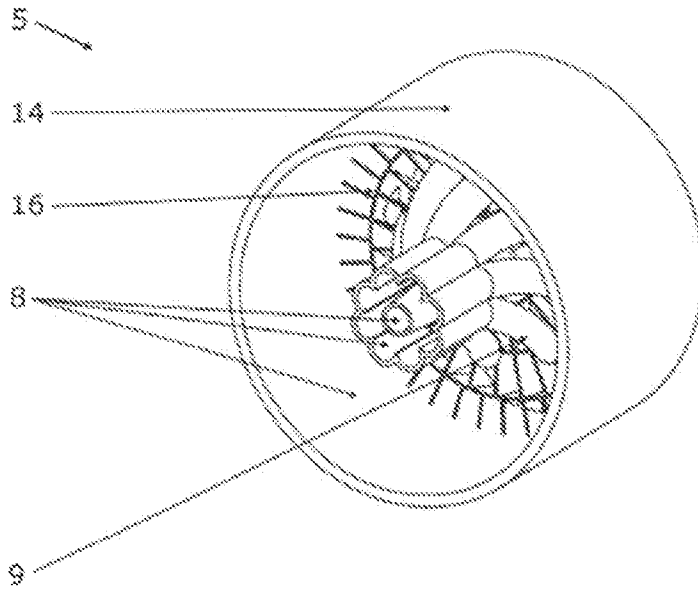


Fig. 7

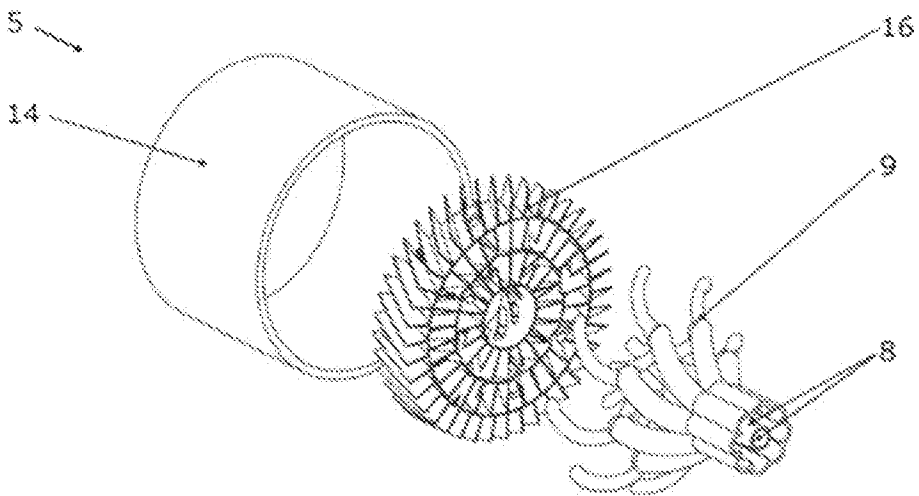


Fig. 8

5/9

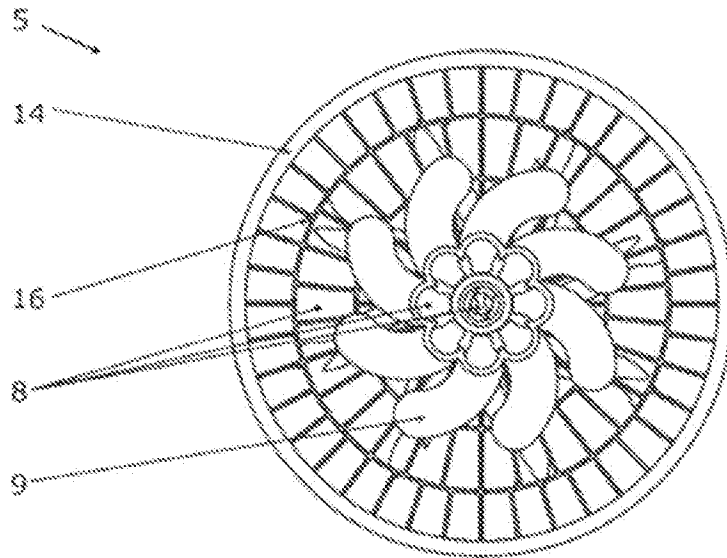


Fig. 9

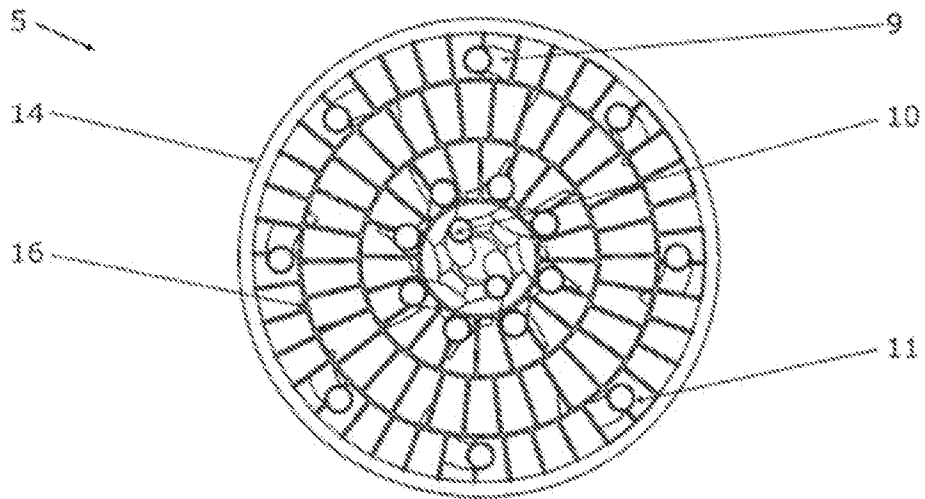


Fig. 10

6/9

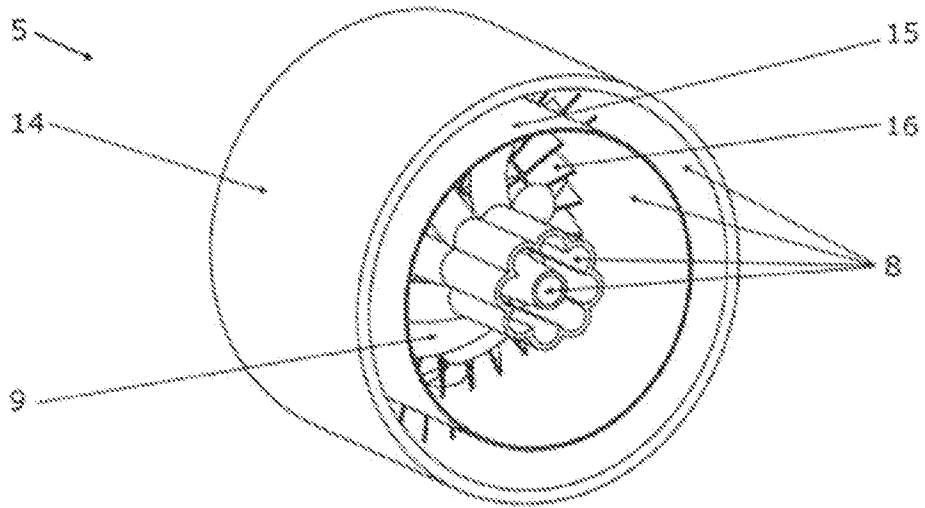


Fig. 11

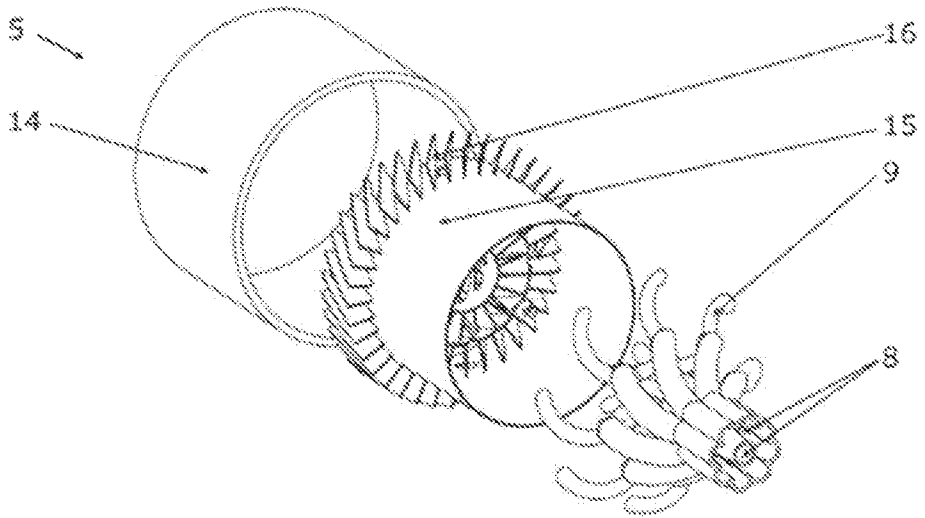


Fig. 12

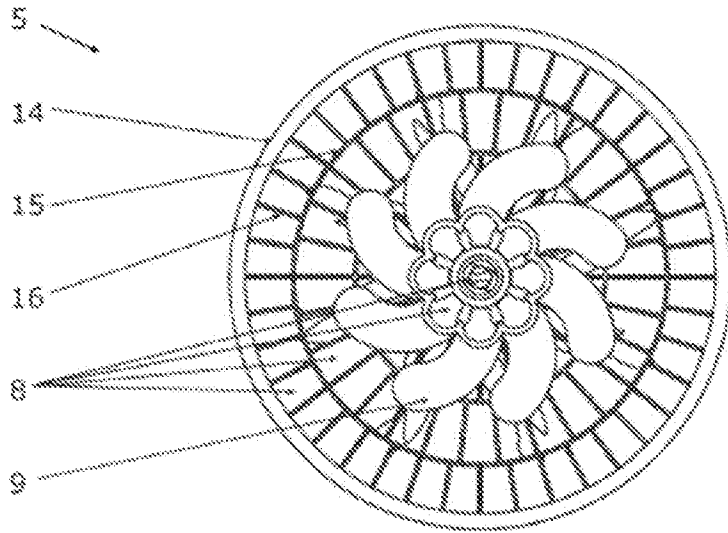


Fig. 13

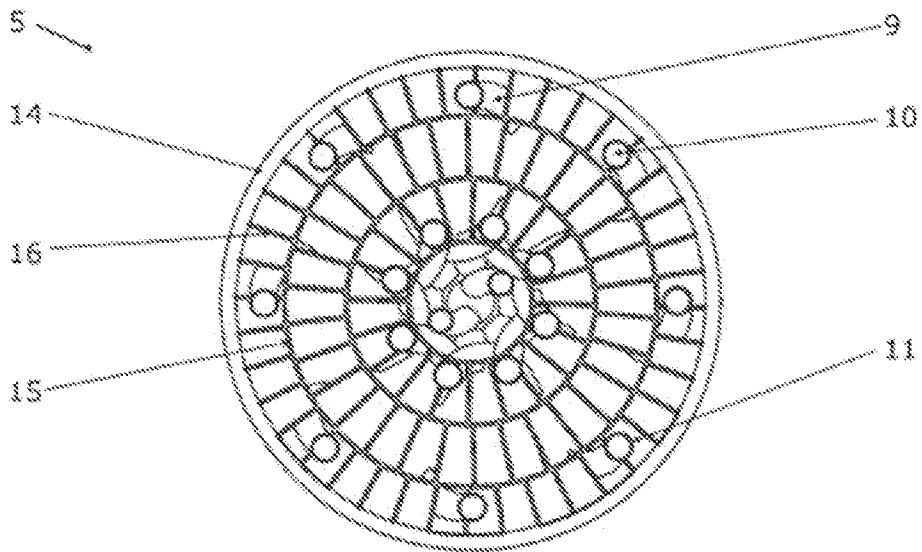


Fig. 14

8/9

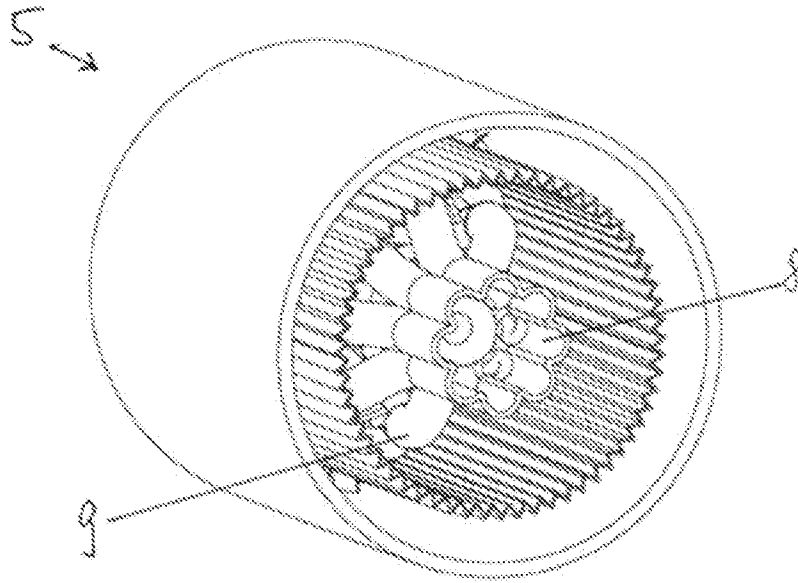


Fig. 15

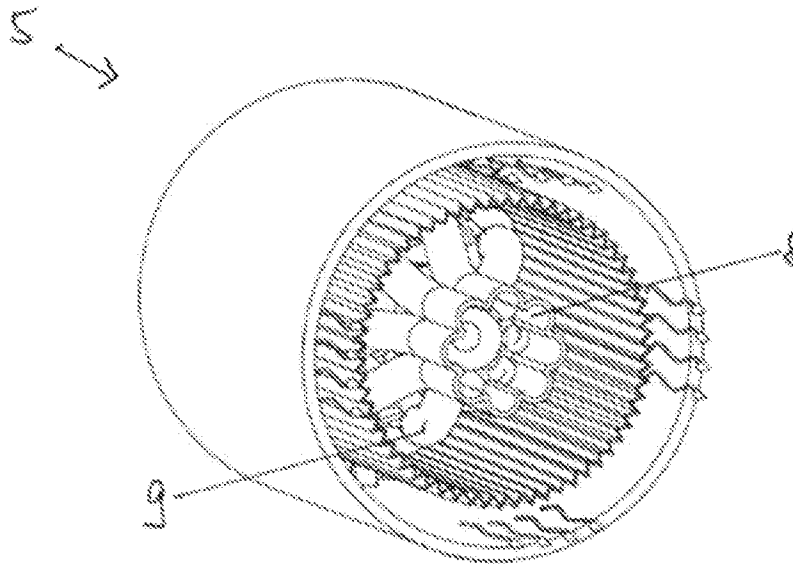


Fig. 16

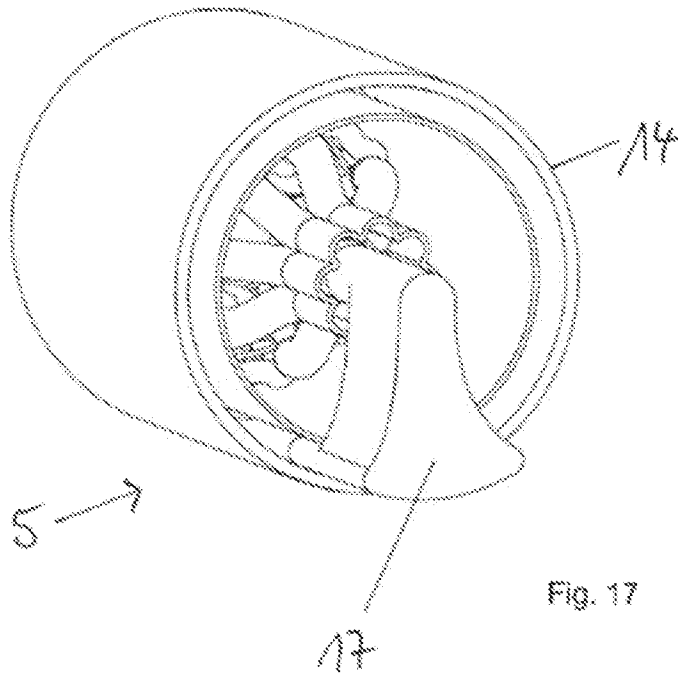


Fig. 17