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- (71) Applicant: **HOLAIRA, INC.** [US/US]; 3750 Annapolis Lane North, Suite 104, Plymouth, MN 55447 (US).
- (72) Inventors: **KAVECKIS, Ryan**; 4632 Grand Avenue South, Minneapolis, MN 55419 (US). **HARSHMAN, Edward, S.**; 11220 Northeast 59th Place, Kirkland, WA 98033 (US). **MAYSE, Martin, L.**; 788-110th Avenue NE, Apt N1401, Bellevue, WA 98004 (US). **STREETER, John**; 27725 Northeast 30th Street, Redmond, WA 98053 (US). **TRUE, Kyle**; c/o Holaira, Inc., 3750 Annapolis Lane North, Suite 104, Plymouth, MN 55447 (US). **GUNDERSON, Richard, C.**; C/o Holaira, Inc., 3750 Annapolis Lane North, Suite 104, Plymouth, MN 55447 (US).

- (74) Agents: **BURGESS, Daidre, L.** et al.; Patterson Thuent Pedersen, P.A., 4800 IDS Center, 80 South Eighth Street, Minneapolis, MN 55402-2100 (US).
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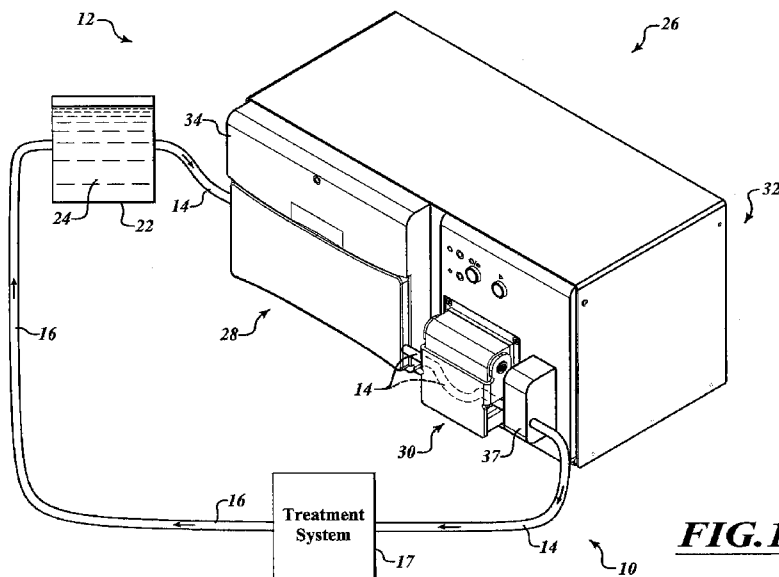


FIG. 1

(57) Abstract: A treatment system includes a fluid cooling supply system for chilling and delivering liquid coolant to a patient. The fluid cooling supply system includes a cooling device and a heat exchanger device. The heat exchanger device is biased to the cooling device and is in fluid communication with a treatment device in a patient. The fluid cooling supply system includes at least one biasing mechanism to provide a given biasing force between the heat exchanger device and the cooling device to effectuate and improve heat transfer. The liquid coolant may be circulated through an energy delivery device positioned in an airway of a patient to preserve tissue. The system is controlled to circulate liquid coolant at a given temperature and pressure for a selected amount of time during pulmonary treatment of a patient.

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FLUID DELIVERY SYSTEM AND METHOD FOR TREATMENT
RELATED APPLICATION

The present application claims the benefit of U.S. Provisional Application No. 61/779,371 filed March 13, 2013, which is incorporated herein in its entirety by reference.

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TECHNICAL FIELD

The present invention generally relates to systems and associated methods for delivering a cooled fluid during treatment of a patient.

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BACKGROUND

DESCRIPTION OF THE RELATED ART

Several conventional medical treatments include supplying a cooled liquid directly to the human body. For example, a cooled liquid may be supplied to the blood stream to cool an organ, such as the brain, to protect the organ from injury.

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Other conventional medical treatments include supplying a cooled liquid to a device used to treat the human body. For example, several particularly effective treatments for pulmonary disorders are described in, for example, U.S. Patent No. 8,088,127, titled, "Systems, Assemblies, and Methods for Treating a Bronchial Tree," and U.S. Patent Application Publication No. 2011/0152855, titled, "Delivery Devices With Coolable Energy Emitting Assemblies." In one example treatment described in these documents, a pulmonary treatment system delivers energy to damage a nerve trunk extending along an airway of a patient. In this example, the energy is delivered to a coolable energy emitter assembly and, simultaneously, chilled fluid is delivered to the energy emitter assembly to cool the energy emitter assembly to avoid or limit destruction of non-targeted tissue.

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Conventional coolant supply systems typically include a pump which pumps a liquid coolant from a reservoir to the patient and/or treatment device. Depending on the type of therapy being performed, conventional liquid coolant supply systems can include relatively large reservoirs containing as much as five gallons of liquid coolant from which liquid coolant is supplied to the thermal therapy catheter. The liquid coolant contained within the large reservoir is, in many cases, simply maintained at room temperature. Other conventional liquid coolant supply systems have closed loop systems in which fluid is pumped from a reservoir and back to the reservoir after circulation through a device in a patient.

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BRIEF SUMMARY

It has been recognized that delivering a liquid coolant to a treatment site in a patient during treatment can present several difficulties to practitioners. For example, it can be challenging to maintain a desired temperature (or range of temperatures) at a treatment site within a patient for a desired interval during a treatment session. This is partially due to heat losses that may exist from the time when the fluid is chilled to the time when the fluid is supplied to a patient for treating the tissue.

It has been recognized that conventional liquid coolant supply systems fail to provide a sufficiently compact and efficient closed loop system that allows for control of the temperature and pressure of a liquid coolant supplied to a treatment device positioned in a patient. In addition, conventional liquid coolant supply systems can be expensive and may, in some instances, require extensive and time consuming sterilization between treatments of different patients. Moreover, it has been recognized that conventional liquid coolant supply systems may not be ideal for use during certain treatments, such as the pulmonary treatment discussed above, because of requirements pertaining to size of an insertion device, temperature at a treatment site, duration of treatment, controllability of the system, and other requirements that may be specific to certain treatments of a patient.

According to one aspect of the present disclosure, a treatment system includes a fluid cooling supply system for treatment of a patient and is configured to chill a fluid and circulate the chilled fluid through a treatment device, such as an energy delivery device, positioned inside a patient. The fluid cooling supply system may include (or be coupled to) a fluid reservoir having a fluid or coolant contained therein. The fluid cooling supply system may include a cooling device having a thermal plate for thermally treating the fluid. A heat exchanger may be removably coupled to the cooling device with a given biasing force for effectuating heat transfer from the fluid contained in or traveling through the heat exchanger. The heat exchanger may be a replaceable or disposable heat exchanger cartridge that includes a thermally conductive surface and a fluid channel that extends through the cartridge. At least a portion of the fluid channel in the cartridge is arranged adjacent the thermally conductive surface. The fluid channel permits passage of the fluid during thermal treatment of the fluid by the cooling device. Accordingly, when the cartridge is coupled to the cooling device, the thermally conductive surface and the thermal plate are biased to each other such that operating the cooling device draws heat from fluid contained in the fluid channel of the cartridge; the chilled fluid may then be supplied to a patient for treatment.

In one aspect, the heat exchange cartridge is a flexible, and preferably disposable, thermoformed tray that is bonded to a plate having a thermally conductive surface. The flexible thermoformed tray includes a recessed serpentine structure defining a fluid channel when the tray is bonded to the plate. A first end of the channel includes an inlet port for coupling to an inlet supply line, and a second end of the channel includes an outlet port for coupling to an outlet supply line. A depth and width of the recessed serpentine structure is determined based on a desired residence time of fluid within the cartridge, the residence time being calculated based on the flow rate of the fluid, and the desired temperature change of the fluid from the inlet to the outlet of the cartridge.

In other aspects, the heat exchanger is a bag removably coupled to the cooling device with a given biasing force for effectuating heat transfer from fluid contained in or traveling through the bag. The bag may be removably coupled to the cooling device by a plate such that the bag is positioned between the cooling device and the plate, or the bag may be attached by other attachment devices. A common feature of effectuating proper heat transfer to achieve a desired fluid temperature is ensuring a given biasing force of the heat exchanger to the cooling device. Thus, the bag may be biased to the cooling device by a clamp, a plate having fasteners, or other such devices. The bag may include a fluid channel that extends through the bag and serpentes throughout. At least a portion of the bag is arranged adjacent the cooling device and is biased thereagainst such that operating the cooling device draws heat from fluid contained in the fluid channel of the bag; the chilled fluid may then be supplied to a patient for treatment.

The fluid cooling supply system may further include a pump to supply and/or circulate a volume of fluid to the patient. At least one controller may be coupled to the cooling device and the pump for regulating the amount heat transfer and fluid volume and pressure to supply to a patient. The fluid cooling supply system may also include a supply path and a return path, which may include a series of lines or tubes or fluid pathways. The supply path originates at the fluid reservoir where the fluid is traversed through a heat exchanger cartridge for chilling the fluid, and then traversed to a treatment device in a patient for cooling at a treatment site. The return path originates at the treatment device in the patient, and then the return path may traverse back to the fluid reservoir for continuous circulation of the fluid through the system. Thus, the fluid reservoir, the supply and return tubes, the fluid channel of the cartridge, and the treatment device are all in fluid communication with each other. Accordingly, the cooling device chills the fluid that the pump circulates throughout the system during treatment of the patient.

As can be appreciated in any aspect of the present disclosure, the fluid cooling supply system may be a closed loop system or an open loop system. In a closed loop system, the fluid is

continuously supplied from and returned to the fluid reservoir for recirculation. In an open loop system, the fluid is supplied from the fluid reservoir to the treatment device and then discarded after circulation through the treatment device.

5 Regarding certain components of the fluid cooling supply system introduced above and according to some aspects, the fluid reservoir may be a bag or other device capable of holding a fluid. In a closed loop system, the fluid reservoir may be a collapsible bag (such as an IV bag used for holding and providing saline or other fluids) having a supply port for providing the fluid and a return port for receiving the fluid once circulated through the system. Using a collapsible bag can, advantageously, accommodate changes in fluid pressure resulting from pumping the
10 fluid through the system from the fluid reservoir, whether pumped in a forward or reverse manner.

In some aspects the fluid reservoir may be fluidly connected to the closed loop system via a coaxial bag spike assembly. The coaxial double spike may include a hypotube inserted within and through a lumen of a female luer, thereby defining a coaxial arrangement of an inner channel
15 and an outer channel. The assembly can further include a bag spike adapter with two ports for connecting the fluid supply and return lines with the inner and outer channels of the assembly. For example, the inner channel is in fluid communication with the return line and the outer channel is in fluid communication with a supply line. In an alternative embodiment, the outer channel is in fluid communication with the return line and the inner channel is in fluid
20 communication with the supply line. Accordingly, the coaxial bag spike assembly precludes the need for a separate supply and return spike by allowing fluid to flow out of and into the fluid reservoir simultaneously. This also allows for standard commercially available IV bags to be utilized as coolant reservoirs.

The cooling device may be any appropriate cooling device, such as a thermoelectric
25 cooler (hereinafter “TEC”), having a thermal plate for effective heat transfer from the fluid when a heat exchanger is coupled to the thermal plate. TECs are typically used for cooling applications and for controlling the amount of heat transfer from a material or fluid, as is well known in the art. TECs use the Peltier effect (or thermoelectric effect) to create a heat flux between the junction of two different types of materials. As such, a typical TEC includes a “hot
30 plate” and a “cold plate” having a plurality of p-type and n-type semiconductors sandwiched between the plates. When a voltage is applied across the semiconductors, the TEC transfers heat from the cold plate to the hot plate, and the heat is dispersed from the hot plate by a heat sink and a fan, for example. Accordingly, the cooling device of the present disclosure is preferably a TEC having a thermal (cold) plate biased against a heat exchanger cartridge to remove heat from the

fluid contained in the cartridge. It will be appreciated that other cooling devices or systems could be used to achieve the same result of cooling the fluid, such as refrigerator system or other cooling system coupled to or including a heat exchanger.

5 The pump is configured to supply and circulate chilled fluid through the treatment device. The pump may be further configured to regulate a volume and a pressure of fluid passing through the system. In some aspects the pump is a peristaltic pump that is coupled adjacent to the cooling device and the cartridge. A peristaltic pump has the capability to draw and push fluid through a tube without contacting the fluid to maintain sterility of the fluid. In a one example, the pump is positioned in a supply path between the heat exchanger cartridge and
10 the patient (or downstream of the cartridge) such that the pump draws the fluid through the cartridge at a negative pressure and supplies chilled fluid to the treatment device at a positive pressure. Positioning the pump downstream of the cartridge in this manner provides several advantages. For example, the resulting negative pressure in the cartridge allows for greater flexibility in material and design choices of the cartridge. Smaller and thinner components can
15 be used in the cartridge, resulting in greater heat transfer from the fluid during system operation. In some aspects, the positive pressure supplied to the energy delivery device is at least 80 psi, and the fluid is returned to the fluid reservoir and/or the cartridge from the treatment device at a pressure of 10 psi or less, although the pressure in the system may vary beyond such values depending upon system and patient requirements.

20 In some aspects, the pump is configured to circulate fluid through the system at a fluid flow rate of between 70 milliliters to 160 milliliters per minute, although the flow rate may vary beyond such range. Preferably, the flow rate is 100 milliliters per minute. In some aspects, the pump is configured to supply chilled fluid to the treatment device at a pressure between 25 psi and 150 psi, although the flow rate may vary beyond such range. Preferably, the pressure is
25 between 80 psi and 100 psi.

In some aspects, the pump includes a forward gear and a reverse gear. The reverse gear is adapted to reverse the flow of the fluid through system to remove gas from the system before or during treatment of a patient. Removing gas or air bubbles from the system allows for an uninterrupted fluid supply during treatment and maximizes cooling of the fluid in the cartridge.
30 The cartridge can also be positioned substantially vertical relative to horizontal and to include an inlet port positioned at an upper portion of the cartridge and an outlet port positioned at a lower portion of the cartridge. With this arrangement, reversing the pump direction will drive fluid back through the system, thereby removing gas from the fluid channel of the cartridge. In particular, the gas rises vertically through the cartridge, and eventually into a fluid reservoir for

dissipation. The pump may then be engaged by its forward gear to supply chilled fluid during treatment of a patient. Even during forward, normal operation of the pump, gas that may exist in the cartridge may tend to rise upwardly due to the particular arrangement and configuration of the cartridge.

5 In some aspects, the fluid channel includes at least one corner portion proximate a transition between a first sidewall and a second side wall of the fluid channel. The at least one corner portion is configured such that gas bubbles are not trapped near or proximate the at least one corner portion during operation of the system. The corner portion may have a radius or chamfer at the transition between the first and second sidewalls of the channel. In addition, the
10 fluid channel may have a cross sectional profile that has a rounded corner portion at upper and lower corners of the cross sectional profile. These features that reduce the cross sectional area of the fluid channel may assist to overcome the surface tension of gas bubbles that may otherwise become stuck in the corners due to the vertical orientation of the cartridge.

In one aspect, the heat exchanger cartridge includes a first plate and a second plate
15 coupled to each other. The first plate includes a thermally conductive surface, which may be comprised of copper, aluminum, and/or stainless steel. The thermally conductive surface is preferably comprised of copper, and more preferably comprised of plated or anodized metals such as anodized aluminum or silver plated copper. The second plate includes a thermally insulating material, such a polymer or plastic, and includes a serpentine groove that defines at
20 least a portion of the fluid channel. The serpentine groove may have a substantially flat profile relative to the thermal plate in order to maximize heat transfer from the fluid. The cartridge may include an input port coupled to a fluid reservoir that supplies fluid, and an output port coupled to the treatment device for supplying chilled fluid. As such, the input and output ports are in fluid communication with the fluid channel and the treatment device. In some aspects, the
25 cartridge includes a variable volume reservoir contained in the cartridge such that fluid is drawn only from the variable volume reservoir and not from any other source. In such aspect, the fluid may then be discarded after circulation through the treatment device (open loop system), or the fluid may be returned to an inlet of the variable volume reservoir (closed loop system). In some aspects, a return fluid channel extends through a portion of the cartridge with at least a portion of
30 the return fluid channel arranged adjacent to the thermally conductive surface such that the fluid in the return fluid channel is pre-cooled before returning to the fluid reservoir for recirculation.

In one aspect, the fluid cooling supply system may include at least one biasing mechanism to provide sufficient and given biasing force between the cartridge and the cooling device. The biasing mechanism can be at least one magnet arranged to removably couple the

cartridge to the cooling device. The at least one magnet may be magnetically coupleable to at least one corresponding magnet adjacent the thermal plate of the cooling device, or it may be magnetically coupleable to a magnetically attractive element of the cooling device. The at least one biasing mechanism may include two pairs of magnets positioned on opposing ends of the cartridge and each coupleable to corresponding pairs of magnets adjacent the thermal plate. The corresponding pairs of magnets may be secured to a biasing frame coupled to the thermal plate of the cooling device. The biasing frame may extend around a perimeter of the thermal plate. The corresponding pairs of magnets of the biasing frame are aligned with and attractable to the pairs of magnets of the cartridge to bias the cartridge to the thermal plate with a given biasing force. The result of utilizing a naturally-occurring means and mechanism is that most or all of the surface area of the thermally conductive surface of the cartridge is biased against the most or all of the surface area of the thermal plate of the cooling device at a given biasing force to effectively and efficiently transfer heat from the fluid during cooling of the fluid.

In several aspects of the present disclosure the cooling system includes features that act to bias the cartridge to the cooling device with a sufficient and given force to effectuate and improve heat transfer from the fluid. Notably, available TECs are limited by the amount of heat flux that is able to be dissipated by the TEC; thus, desirable heat transfer of the fluid is somewhat limited in some applications. Also, TECs are known to be somewhat inefficient as compared to other cooling devices, so it is important to reduce efficiencies of the system in other aspects, such as the design of the cartridge and configuration of other components in the system, like the position of the pump. Furthermore, a sufficient biasing force between the thermally conductive surface of the cartridge and the thermal plate of the cooling device is important because of the nature of the material of the surfaces biased to each other. The thermally conductive surface may be copper and thermal plates are typically a ceramic substrate. Under a microscope, even the smoothest of copper and ceramic surfaces show countless ridges and valleys that may affect thermal conductivity between the two materials if a sufficient biasing force is not applied and maintained during heat transfer. According, the present disclosure provides an effective means and various mechanisms to adequately bias the cartridge to the cooling device to increase surface-to-surface contact between the biased surfaces to efficiently chill the fluid during treatment of a patient. Such improved surface contact ultimately reduces heat losses in the system so that a constant and controllable fluid temperature is supplied to a treatment device in a patient. This is of particular importance when operating the cooling system during pulmonary treatment, which requires, at certain intervals, a constant fluid temperature and a constant fluid pressure for a particular duration during a treatment session.

In one aspect, the cartridge may be formed and provided in a pre-stressed configuration to improve heat transfer and reduce heat losses. Accordingly, the cartridge may be manufactured to be in a first state (pre-stressed) when disengaged from the cooling device, and in a second state when engaged to the cooling device. The first state is achieved by forming the cartridge to have a profile with a convex shape relative to the thermal plate of the cooling device such that a lateral arc of the cartridge extends from a left side to a right side of the cartridge. Accordingly, when the cartridge is engaged to the thermal plate (i.e., by utilizing the pairs of magnets on left and right sides of the cartridge, for example), by virtue of the convex shape and the force of the magnets, the thermally conductive surface of the cartridge will have a profile that is a substantially flat shape relative to the thermal plate because the magnets on the sides of the cartridge will tend to “flatten out” the profile of the cartridge. This pre-stressed configuration tends to prevent a slight “buckling” that may be experienced by the cartridge such that the result would be a concave cartridge that is not completely or adequately biased to the cooling machine. Thus, the pre-stressed configuration of the cartridge provides greater surface-to-surface contact between the thermally conductive surface and the thermal plate, thereby resulting in improved heat transfer while reducing heat losses in the system. This is of particular importance when operating the cooling system during treatment of a patient because this particular pulmonary treatment requires, at certain intervals, a constant fluid temperature and a constant fluid pressure for a particular duration during a treatment session.

A method is provided for attaching and removing a heat exchanger cartridge from a cooling system for treatment of a patient. In some aspects, the method includes biasing a heat exchanger cartridge to a thermal plate of a cooling device, such as the cartridge and cooling device having the same or similar features discussed in the present disclosure. The method includes removing the heat exchanger cartridge from the cooling device, which may occur after treatment of one or more patients or treatment sessions. The method includes biasing a replacement heat exchanger cartridge to the thermal plate of the cooling device. The step of biasing the cartridges may include engaging magnets or other biasing mechanisms such that a given biasing force is applied to the cartridge to effectuate efficient heat transfer from the fluid. In preferred configurations, the given biasing force is at least 10 pounds of force, and is between 10 and 60 pounds of force, but the given biasing force may vary beyond such values and ranges. The given biasing force provided by the magnets permits biasing of the thermally conductive surface of the cartridges to the thermal plate of the cooling device. Because of the configuration of the magnets, biasing the cartridge to the cooling device occurs automatically such that the cartridge is positioned at approximately the same position on the cooling device with each

replacement cartridge. This provides one advantage of a system that maintains consistency of position for every replaceable cartridge coupled to a cooling device, and therefore provides consistency of efficiency of chilling the fluid in the cartridge with repeated uses of system and replacement cartridges. The method may further include pumping the fluid through the heat exchanger cartridge for delivery to the patient before removing the heat exchanger cartridge from the cooling device. The method may further include supplying chilled fluid to a treatment device (e.g., energy delivery device) positioned adjacent to pulmonary tissue of the patient during a pulmonary treatment.

In another aspect, the fluid cooling supply system may include a cooling device with a thermal plate for cooling fluid, a disposable heat exchanger cartridge removably coupled to the thermal plate, and at least one biasing mechanism coupled to the cartridge and the cooling device to transfer heat from the fluid contained in the cartridge. The cartridge may include a first plate and a second plate coupled to each other wherein the first plate includes a thermally conductive surface, such as copper, aluminum, and/or stainless steel, and the second plate includes a thermally insulating material, such as polymer, ABS, nylon, or polycarbonate. The second plate may include a serpentine groove defining a fluid channel, similar to the cartridge discussed with reference to the magnetically attractable cartridge. In one configuration, the cartridge includes an upper angled surface and a corresponding lower angled surface to be received into a front plate for biasing to the cooling device. The cartridge may include a handle at an end of the cartridge for easy removal and replacement of the cartridge. A backside of the second plate may include a plurality of recesses for improved heat transfer of the fluid via the cooling device.

The cartridge may include a fluid reservoir for supplying fluid through the system; the fluid reservoir may be wholly contained in the cartridge or may be coupled to an outer portion of the cartridge. Accordingly, the second plate includes a fluid reservoir in fluid communication with the fluid channel and positioned at an upper portion of the cartridge. The fluid reservoir in this aspect may be a collapsible bag positioned in a cavity in the second plate. The fluid is supplied from the fluid reservoir to a treatment device and may be either returned to the fluid reservoir in a closed loop system, or discarded as waste in open loop system. Providing a fluid reservoir inside of the cartridge itself provides an advantage of reducing the number of components and steps to set up and operate the system, which ensures sterility of the fluid as it reduces risk of human error due to incorrect installation or use of unsterile components. It also provides an advantage that the fluid in the reservoir is cooled by cooling device during operation, as opposed to providing room temperature fluid from an external fluid reservoir.

The at least one biasing mechanism may be a cam system that has a first position for engaging the cartridge to the cooling device and a second position for disengaging the cartridge from the cooling device. As further discussed in the present disclosure, providing a biasing mechanism (such as this cam system) provides an effective means to adequately bias the thermally conductive surface of the cartridge against the thermal plate of the cooling device with a sufficient and given force in order to increase surface-to-surface contact between the cartridge and the cooling device. In some aspects, a front plate is coupled to the front of a housing containing the cooling device. The cam system, the front plate, and the cartridge operate together to bias the cartridge to the thermal plate. The front plate includes an opening to receive the thermal plate of the cooling device and to facilitate biasing of the cartridge to the thermal plate. The front plate may have a slot sized to slideably receive the cartridge. The slot of the front plate includes an upper biasing surface and a lower biasing surface. The upper and lower biasing surfaces are each non-parallel to the thermal plate and may correspond to the upper and lower angled surfaces of the cartridge. Thus, the slot may have a trapezoid-shaped cross sectional profile that corresponds to a trapezoid-shaped cross sectional profile of the cartridge. Accordingly, the cartridge may be slideably receivable in the slot of the front plate when the cam system is disengaged (or unlocked). Once the cartridge is positioned in the slot, the cam system may be engaged (or locked) to apply a given biasing force to the cartridge against the cooling device to effectuate cooling of the fluid during operation of the system.

In some configurations, the cam system includes a cam lever, a cam shaft having at least one cam lobe, an actuation member coupled to the cartridge, and at least one actuation device coupled to the actuation member and coupleable to the cam lobes. The cam lever is either directly attached to the cam shaft or dynamically linked to the cam shaft. In some configurations, four cam lobes are formed along a length of the cam shaft and spatially separated from each other, although the four cam lobes may be a single cam lobe or cam device. Corresponding to the position of the four cam lobes may be four actuation devices, coupled to the actuation member, and positioned adjacent respective cam lobes. The four actuation devices are actuated downwardly by the respective cam lobes when the cam shaft is rotated by movement of the cam lever from the disengaged state to the engaged state. The actuation member has a lower actuation surface that may be formed at an angle that may correspond to the angle of the upper angled surface of the cartridge. Thus, engaging the cam system will bias the lower actuation surface to the upper angled surface of the cartridge, which tends to force the cartridge slightly downwardly and inwardly toward the cooling device because of the trapezoid-shaped profiles of the slot and the cartridge and the angle of the lower actuation surface, which

collectively tend to bias the cartridge against the cooling device in a lateral direction with a given biasing force when the cam system is engaged.

5 A method is provided to provide a replaceable heat exchanger cartridge to a cooling device utilizing a cam system. The method may include biasing the cartridge to the cooling device by actuating the cam system to an engaged state. The method may include actuating the cam system to a disengaged state to release the biasing force on the cartridge. The method may include removing the cartridge and replacing it with a replacement cartridge, which may be biased to the cooling device with the cam system during treatment of a patient.

10 In another aspect, the at least one biasing mechanism may be a hinged door hingedly coupled to the cooling device and biased towards a closed position, thereby sandwiching the cartridge between the hinged door and the cooling surface of the cooling device. In this embodiment, one or more magnets may be employed on opposing surfaces of the door and/or the cooling device to provide a sufficient force in order to increase surface-to-surface contact between the cartridge and the cooling device. In one aspect, the cartridge may be configured with one or more notches defined in at a least one side edge of the cartridge for alignment with one or more keys of the cooling device to ensure that the cartridge is inserted in an orientation that allows for normal operation.

15 According to some aspects of the present disclosure, a method of cooling fluid for treatment of a patient is provided. The method may include drawing a coolant through a heat exchanger at a negative pressure to chill the coolant. The method may further include positioning a treatment device inside a bronchus of the patient and supplying the coolant to the treatment device to transfer heat from the patient during treatment. The method may include supplying the coolant from a reservoir and returning the fluid to the reservoir in a closed loop system. Alternatively, the method may include supplying the coolant from a reservoir and disposing of the fluid after transferring heat from the patient to the fluid in an open loop system. The method may include supplying the fluid to the treatment device in a positive pressure. The method may include regulating the amount of heat transfer from the coolant with a controller coupled to a cooling device, and regulating the amount of volume of fluid to supply for treatment of the patient with a controller coupled to a pump.

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30 According to some aspects of the present disclosure, a method of cooling fluid for treatment of a patient is provided. The method may include positioning a heat exchanger against a cooling device. The heat exchanger may comprise some or all of the features of the cartridges discussed in the present disclosure. The method may include positioning the heat exchanger in a substantially vertical orientation such that gas rises in the heat exchanger. The method may

include positioning a pump at a downstream side of the heat exchanger and pumping the fluid through the heat exchanger in a reverse manner to substantially remove gas from the heat exchanger and a system. The method may further comprise some or all of the steps for providing cooled fluid to a patient as discussed in the present disclosure.

5 In some aspects according to the present disclosure, a system for treatment of a patient is provided. The system may include a fluid cooling supply device configured to draw a fluid through a heat exchanger at a negative pressure to chill the fluid and to deliver the chilled fluid to the patient at a positive pressure. The fluid cooling supply device may include some or all of the features discussed in the present disclosure, such as the cooling device, pump, controller,
10 housing, and front plate. Likewise, the heat exchanger may include some or all of the features of the cartridges discussed in the present disclosure. The system may include an energy delivery device positioned in the patient and coupled to the fluid cooling supply device such that the fluid cooling supply device circulates the chilled fluid through the energy delivery device to cool the energy delivery device during treatment of the patient. The energy deliver device may include
15 an electrode adapted to deliver energy to a target tissue of the patient. The energy deliver device may include a cooling member arranged adjacent the electrode. The cooling member may be configured to allow circulation of the fluid from the fluid cooling supply device. The electrode and the cooling member are arranged adjacent to a wall of an airway of the patient such that the delivery of energy to the electrode and circulation of chilled fluid through the cooling member
20 damages nerve tissue so that nervous system signals in the patient are attenuated while preserving tissue. The system may include a pump downstream from the cooling device and configured to circulate the fluid to through the energy delivery device at the positive pressure. The method may further comprise some or all of the steps for providing cooled fluid to a patient discussed in the present disclosure.

25 In some aspects of the present disclosure, the temperature of the fluid supplied by the fluid cooling supply system (or by any other system and method described in the present disclosure) may be provided at a given temperature or range at the location of the energy delivery device or other treatment device. It is preferred that the temperature at the energy delivery device energy is maintained at or below 20°C during treatment of the patient. In a
30 preferred configuration, the temperature at the energy delivery device is maintained between 20°C and -5°C during treatment of the patient. In an even more preferred configuration, the temperature at the energy delivery device is maintained between 5°C and -2°C during treatment of the patient. The temperature may vary beyond such ranges, however, depending upon system and patient requirements. In some aspects, a fluid is supplied to a patient at a given temperature

for a selected amount of time during a treatment portion of treatment of the patient, or during an entire treatment process of the patient. In some configurations, the selected amount of time for a particular treatment portion is up to 120 seconds to provide the fluid having a given temperature. In some configurations, the selected amount of time for a particular treatment portion is less than
5 60 seconds. In some configurations, the selected amount of time for a particular treatment portion is between 60 and 120 seconds to provide the fluid having a given temperature. In some configurations, the selected amount of time for a particular treatment portion at least 120 seconds to provide the fluid having a given temperature. The selected amount of time may vary beyond such values and ranges, however, depending upon system and patient requirements. In some
10 configurations, the fluid contained in the heat exchanger cartridge may be cooled to a temperature of at least 20°C, upon exiting the cartridge, and more preferable the fluid is cooled to a temperature between 5°C and -5°C upon exiting the cartridge, although the temperature of the fluid in the cartridge may vary beyond such values and ranges.

In some aspects, a method of treating a patient is provided. The method may include
15 providing a cooling device having a fluid heat exchanger to deliver fluid to the patient, such as the cooling devices and heat exchanger cartridges discussed in the present disclosure. The method may include positioning an ablation assembly of a delivery device within an airway of the patient such that the ablation assembly is apposed against a wall of the airway. The ablation assembly may include an electrode adapted to deliver energy. The method may include coupling
20 a fluid heat exchanger to the ablation assembly to be in fluid communication with each other. The method may include chilling the fluid in the fluid heat exchanger with a cooling device and treating tissue by circulating the fluid from the fluid heat exchanger through the delivery device. The method may include, simultaneously, delivering energy from the electrode of the ablation assembly to treat tissue adjacent the airway of the patient. As such, the method may include
25 damaging nerve tissue of a nerve trunk adjacent the airway such that nervous system signals transmitted to a portion of the bronchial tree are attenuated. As discussed in the present disclosure, the fluid may be drawn through the fluid heat exchanger at a negative pressure and supplied to the delivery device at a positive pressure. During treatment, the fluid in the heat exchanger is cooled by the cooling device at a given temperature, and the fluid is supplied to (or
30 circulated through) the delivery device at a given temperature and for a selected amount of time, as further discussed in the present disclosure.

As will be appreciated by a person having ordinary skill in the art reviewing this disclosure in detail, the methods and systems pertaining to the fluid cooling supply systems and heat exchanger cartridges can be combined in various aspects while still achieving the result of

circulating chilled fluid through a treatment device positioned in a patient during treatment of the patient, as further discussed herein.

BRIEF DESCRIPTION OF THE SEVERAL VIEWS OF THE DRAWINGS

5 Figure 1 is a perspective view of a treatment system having a fluid cooling delivery system according to one aspect.

Figure 2 is a partially exploded view of a fluid cooling delivery system according to one aspect.

Figure 3 is a schematic illustration of a fluid cooling delivery system coupled to a patient.

10 Figure 4 is a front view of a fluid cooling delivery system and a schematic illustration of a treatment system during a treatment session according to one aspect.

Figure 5 is a schematic illustration of a fluid cooling delivery system coupled to an energy delivery device positioned in a patient during a treatment session according to one aspect.

15 Figure 5A is a cross-sectional view of the supply and return lumens of the treatment device of Figure 5, taken along lines 5A-5A of Figure 5.

Figure 6A is an isometric view of a heat exchanger cartridge according to one aspect.

Figure 6B is an exploded view of a heat exchanger according to one aspect.

Figure 6C is an exploded view of a heat exchanger according to one aspect.

20 Figure 6D is cross sectional side view of the heat exchanger of Figure 6A, taken along lines 6D-6D of Figure 6A.

Figure 6E is a cutout view of a portion of the heat exchanger of Figure 6D.

Figure 7A is a side view of a heat exchanger according to one aspect, showing the cartridge in a first state disengaged from a cooling device.

25 Figure 7B is a side view of the heat exchanger of Figure 7A, showing the heat exchanger in a second state and engaged to a cooling device.

Figure 8 is a perspective view of a fluid cooling delivery system during a treatment session according to one aspect.

Figure 9 is a partially exploded view of a fluid cooling delivery system according to one aspect.

30 Figure 10 is a schematic illustration of a fluid cooling delivery system during a treatment session according to one aspect.

Figure 11A is a back side perspective view of a portion of a fluid cooling delivery system according to one aspect, showing a cam system disengaged and a heat exchanger removed.

Figure 11B is the back side perspective view of Figure 11A, showing the cam system engaged and the heat exchanger installed.

Figure 12A is side view of the front plate of Figure 11A according to one aspect.

5 Figure 12B is a side cross sectional view of a portion of a fluid cooling delivery system of Figure 11B, taken along lines 12B-12B of Figure 11B.

Figure 13A is an isometric view of the heat exchanger cartridge of Figure 11A.

Figure 13B is an inside perspective view of the heat exchanger of Figure 11A.

Figure 13C is a cross sectional view of the heat exchanger of Figure 13A, taken along lines 13C-13C of Figure 13A.

10 Figure 14 is a perspective view of a treatment system having a fluid cooling delivery system according to one aspect.

Figure 15 is a front view of a fluid cooling delivery system and a schematic illustration of a treatment system during a treatment session according to one aspect.

15 Figure 16 is a partially exploded view of a fluid cooling delivery system according to one aspect.

Figure 17 is back perspective view of a hinged door assembly according to one aspect.

Figure 18 is a schematic illustration of a fluid cooling delivery system coupled to an energy delivery device positioned in a patient during a treatment session according to one aspect.

20 Figure 19A is a front perspective view of a heat exchanger cartridge according to one aspect.

Figure 19B is a back perspective view of a heat exchanger cartridge according to one aspect.

25 Figure 20 is a back perspective view of a coaxial double spike assembly according to one aspect.

DETAILED DESCRIPTION

30 According to the present disclosure, Figures 1-7B illustrate a first aspect of a treatment system having a fluid cooling supply system for treatment of a patient, Figures 8-13C illustrate a second aspect of a treatment system having a fluid cooling supply system for treatment of a patient, and Figures 14-20 illustrate a third aspect of a treatment system having a fluid cooling supply system for treatment of a patient. It will be understood that various configurations described with reference to the first and second aspects may be combined into further configurations and aspects, which may be further discussed in the present disclosure regarding particular configurations.

Figures 1 and 2 illustrate a system 10 that includes a fluid cooling supply system 12 coupled to a treatment system 17. Figure 2 shows a partially exploded view of certain components of the fluid cooling supply system 12 of Figure 1.

In the example in Figure 1, the fluid cooling supply system 12 is coupled to the treatment system 17. The treatment system 17 may be positionable at least partially in a patient (Figure 4). The fluid cooling supply system 12 is configured to chill a fluid, pump the fluid, and supply the fluid through the treatment system 17. In a closed loop system, the fluid cooling supply system 12 may include a fluid reservoir 22, a fluid 24, a cooling system 26, a heat exchanger cartridge 28, and a supply line 14 and a return line 16, which collectively cooperate to circulate cooled fluid through the treatment system 17 during treatment. The supply line 14 originates at the fluid reservoir 22 and extends through the cartridge 28 and along a pump 30. The supply line 14 may extend through a pulse damper 37 for damping vibration of the supply line 14 during operation of the pump 30. Finally, the supply line 14 extends into the treatment system 17 positionable in a patient. The return line 16, in fluid communication with the supply line 14, originates at the treatment system 17 and extends from inside the patient and back to the reservoir 22 for recirculation of the fluid during treatment.

Figure 2 further shows an exploded view of portions of the cooling system 26. The cooling system 26 may include a housing 32, a cooling device 36, a heat exchanger cartridge 28, a controller 42, and a pump 30. The housing 32 includes a first portion 31, a second portion 33 and a front plate 34 coupled to the first portion 31. The first portion 31 and second portion 33 of the housing 32 are removably attached to each other and are configured to structurally support and house various components of the system. The cooling device 36 has a thermal plate 38 extending at least partially through the front plate 34. The front plate 34 is secured to a front area of the first portion 31 of the housing 32. The front plate 34 and the housing 32 cooperate to structurally support the cooling device 36 and to position the thermal plate 38 substantially vertical. The front plate 34 includes an opening 34a for receiving the thermal plate 38 of the cooling device 36 and for facilitating biasing of the cartridge 28 to the thermal plate 38 (Figure 7B). The housing 32 further includes a spacer 40 positioned between the cooling device 36 and the front plate 34 for additional support of the cooling device 36 and to allow egress of the thermal plate 38 through the front plate 34.

The cooling device 36 may be, for example, a conventional TEC that includes the thermal plate 38, a hot plate 39, fins 46, and a fan 48. The front portion 31 of the housing includes an opening 35 for receiving the cooling device 36 such that the thermal plate 38 extends out of the housing 32. A support plate 47 of the cooling device 36 may be secured to the first

portion 31 of the housing 32 to properly position the cooling device 36. The support plate 47 may further be secured to the spacer 40 and front plate 34 for additional structural support.

The spacer 40 is coupled between the front plate 34 and the cooling device 36. The spacer 40 includes an opening 40a to allow egress of the thermal plate 38 and to position the thermal plate 38 adjacent the opening 34a of the front plate 34. The spacer 40 extends around a perimeter of the thermal plate 38 and the hot plate 39. Accordingly, an outer surface 49 of the spacer 40 and a planar surface 51 of the thermal plate 38 are substantially planar to each other (Figure 7B) so that the cartridge 28 may be biased to the thermal plate 38.

The heat exchanger cartridge 28 includes a first plate 41 and a second plate 43. Magnets 99 are positioned in the second plate 43 (Figure 6A). The spacer 40 includes four magnets 53 positioned at corresponding positions to engage the magnets 99 of the heat exchanger cartridge 28. Thus, the magnets 99 of the second plate 43 are magnetically coupled to the magnets 53 in the spacer 40 to removably couple the cartridge 28 to the thermal plate 38. Thus, the first plate 41 is biased to the planar surface 51 of the thermal plate 38 with a given biasing force to effectuate heat transfer of fluid contained in the cartridge 28 (Figures 6A-6C and 7B).

A controller plate 55 may be secured to a front area of the first portion 31 of the housing 32. The controller plate 55 may include an opening 57 for receiving the pump 30. The pump 30 may be a peristaltic pump having a cover 50 and a rotating device 59 for coupling to the supply line 14, such as with available peristaltic pumps. The fluid supply tube is placed in the pump in contact with the rotating device. The cammed surfaces on the rotating device cause periodic pressurization of the fluid in the fluid supply line. The pump 30 can include clamping mechanisms on the upstream and downstream sides thereof to ensure that the fluid supply line does not get pulled into the rotating device when the pump direction is reversed. In the present example, the pump 30 is positioned downstream of the cartridge 28 such that the cartridge 28 experiences a negative fluid pressure and the treatment system 17 experiences a positive fluid pressure during normal operation of the treatment system.

A pulse damper 37 may be removably attached to the controller plate 55. The damper 37 can be, for example, a chamber that includes an inlet and an outlet. The chamber accumulates a volume of fluid immediately downstream of the pump. The damper acts in a manner similar to a capacitor in a signal filtering device in that it smoothes the pressure oscillations generated by the rotating device of the pump.

A controller system 60 includes control devices 62 and the controller 42 for controlling fluid temperature, pressure, and velocity. The control devices 62 are provided on the controller plate 55 and are coupled to the controller 42. A practitioner may operate the control devices 62

to control the system. The controller 42 may be operably coupled to the pump 30 to regulate the speed and direction of the pump 30, thereby regulating the direction of flow and amount of volume of fluid circulating through the system (Figure 3). The controller 42 may also be operably coupled to the cooling device 36 for regulating the temperature of the fluid in the cartridge 28, thereby regulating the temperature of the fluid circulating through the treatment system 17, and thereby further regulating the temperature of a treatment device and/or patient tissue (Figure 4). Performance can be optimized based on feedback from sensors that detect fluid and tissue temperatures, tissue impedance, and fluid supply to the treatment device (*e.g.*, a pressure sensor, a temperature sensor, a thermocouple, a contact sensor, or the like). Accordingly, if surface temperature of patient tissue becomes excessively hot, fluid cooling can be increased by the cooling device 36 and/or electrode power can be decreased in order to produce deep lesions while protecting surface tissues.

Figure 3 is a schematic illustration of a treatment system 101 according to one aspect of the present disclosure. The treatment system 101 includes a fluid cooling supply system 12 having a cooling system 26, a heat exchanger 28, and a fluid reservoir 22. The fluid cooling supply system 12 includes a cooling device 36, a controller 42, and a pump 30. The heat exchanger 28 is coupled to the fluid reservoir 22, the cooling device 36, and the pump 30. A supply path 66 and a return path 68 extend from the fluid cooling supply system 12 and are coupled to a treatment device 20 which may be positioned inside a patient 64. The supply path 66 originates at the fluid reservoir 22, extends through the heat exchanger 28, then through the pump 30 before extending into the patient 64 and coupled to the treatment device 20. The return path 68 originates at the treatment device 20 and terminates at the fluid reservoir 22 for recirculation of the fluid through the system 101. Alternatively, the return path 68 may be coupled to a waste reservoir 67 in an open loop system.

In the illustrated example, the pump 30 draws fluid from the fluid reservoir 22 and through the heat exchanger 28 at a negative pressure. The fluid is chilled by the cooling device 36 as it travels through the heat exchanger 28. The fluid is then supplied to the treatment device 20 by the pump 30 at a positive pressure via the supply path 66. The fluid is circulated through the treatment device 20 and returned from the treatment device 20. In some aspects, the heat exchanger 28 may include the fluid reservoir 22 inside the fluid exchanger 28 (Figure 6C).

In this example, the pump 30 includes a forward gear and a reverse gear, as depicted by arrows P. The forward gear draws the fluid from the fluid reservoir 22 and through the heat exchanger 28 to circulate chilled fluid through the treatment device 20. Conversely, the reverse gear pushes the fluid in reverse through the heat exchanger 28 to expel gas that may exist in

portions of the system 101. In some aspects, the pump 30 is coupled to a controller for variable control over the speed of the pump in order to control the amount of fluid delivery to the treatment device. Thus, the size and apposition pressure of the treatment device may be controlled by the variable speed controller. Moreover, a non-contact pressure measurement device may be electrically coupled to the pump and positioned proximate the high pressure side of the fluid path to regulate system pressure, such as by varying the speed of the pump in response to the pressure measured by the non-contact pressure measurement device, for example.

In some aspects, a pump 30a is provided downstream of the treatment device 20 to draw the fluid from the treatment device 20. Accordingly, the pump 30 and the supplemental pump 30a cooperatively act to circulate chilled fluid through the system. The pump 30a may draw up to 14 psi of fluid pressure from the treatment device 20. Accordingly, the pressure downstream of the treatment device 20 may be lower, such as around 10-20 psi, while the pressure upstream the treatment device 20 may be higher, such as around 80-100 psi. Such configuration of providing an additional pump downstream a treatment device improves cooling at the treatment region in the patient because the flow rate is increased through the treatment device by virtue of simultaneously pushing the fluid with one pump while drawing the fluid with another pump. Furthermore, by drawing fluid from the treatment device 20 via the pump 30a, a lower fluid pressure may be exhibited in the treatment device 20 than without such additional pump. In some aspects, a pump 30a is the only pump or device circulating fluid through the system. Such configuration can further lower fluid pressure downstream of the treatment device.

Figure 4 shows a treatment system 201 according to one aspect of the present disclosure. The treatment system 201 may include a fluid cooling supply system 12 and a pulmonary treatment system 19. The fluid cooling supply system 12 may be coupled to the pulmonary treatment system 19 by a supply line 14 and a return line 16. The pulmonary treatment system 19 may include a flexible bronchoscope 18 having a control portion 68, a steering mechanism 70, and a video system 72. The flexible bronchoscope 18 may include an insertion tube 74 extending from a control section 76 external to the patient's body, through the trachea 78, and to a treatment device 20 at a treatment site within the left main bronchus 80 of the lungs 81 of a patient. The treatment device 20 can be positioned in the left main bronchus 80, or positioned in other locations, such as within the right main bronchi, the lobar bronchi, and bronchus intermedius. The treatment device 20 can be navigated through tortuous airways to perform a wide range of different procedures, such as, for example, denervation of a portion of a lobe, an entire lobe, multiple lobes, or one lung or both lungs. In some embodiments, the lobar bronchi

are treated to denervate lung lobes. Based on the effectiveness of the treatment, the physician can concurrently or sequentially treat additional lobe(s).

The steering mechanism 70 may be coupled to the bronchoscope 18 and may receive the supply line 14 and the return line 16 to allow egress of the lines into the bronchoscope 18 and ultimately to the treatment device 20 (Figure 5). The bronchoscope 18 may be coupled to the video system 72, which allows a practitioner to observe progress of the insertion tube 74 through the patient on a monitor 82 as the insertion tube 74 is steered with the assistance of the control portion 68. The video system 72 can allow a practitioner to determine whether fluid is supplied to the treatment device 20. The bronchoscope 18 may be coupled to the control portion 68 to control some or all aspects of treatment, such as the amount of energy delivered to the treatment device 20.

The fluid cooling supply system 12 may have the same or similar features as with the systems described with reference to Figures 1-3. The supply line 14 of the fluid cooling supply system 12 originates at a fluid reservoir 22 and through a heat exchanger 28 and through a pump 30. The supply line 14 extends through a damper 37 and then through the steering mechanism 70 for fluid supply to the treatment device 20. The return line 16 originates at the treatment device 20 and extends from the steering mechanism 70 and back to the fluid reservoir 22. Accordingly, the pump 30 may draw fluid from the fluid reservoir 22 and through the heat exchanger 28 while the fluid is chilled by the cooling device 36 (Figure 3). The fluid may travel through a fluid channel 114 of the heat exchanger 28. The fluid may then be supplied to the treatment device 20 at a positive pressure via the supply line 14. The fluid may be circulated through the treatment device 20 and returned from the treatment device 20 to the fluid reservoir 22 in a closed loop system. The cooling device 36 and the pump 30 may be manually controlled by the controller devices 62.

Figure 5 shows a treatment system 301 according to an aspect of the present disclosure. The system 301 includes a fluid cooling supply system 12 coupled to a treatment device 20' for circulating fluid through the treatment device 20' positioned in a patient. For purposes of illustration, the treatment device 20' is shown in a side elevation view positioned in a bronchus 80. By way of example, the schematic of the fluid cooling supply system 12 of Figure 3 is shown having a supply path 66 and a return path 68 in fluid communication with the treatment device 20'. The fluid cooling supply system 12 is not described in detail with reference to Figure 5 as it may include some or all of the same features as described with reference to Figure 3 and with reference to Figure 8, for example.

In some aspects, the treatment device 20' includes an expandable member 82 that extends from a distal end of an elongate member 91. Figure 5A shows a cross sectional view of the elongate member 91 taken along lines 5A-5A. The elongate member 91 may include a supply lumen 93 and a return lumen 95. The supply lumen 93 is in fluid communication with the supply path 66 of the fluid cooling supply system 12, and the return lumen 95 is in fluid communication with the return path 68. A fluid supply channel 97 also extends from the distal end of the elongate member 91, around a portion of the circumference of the expandable member 82, to a distal end of the expandable member 82. A proximal end of the fluid supply channel 97 is in fluid communication with the supply lumen 93, and a distal end of the fluid supply channel 97 is in fluid communication with the interior of the expandable member 82. The return lumen 95 is in fluid communication with the interior of the expandable member 82 at a proximal end of the expandable member 82. The return lumen 95 may surround the supply lumen 93 in the elongate member 91. The fluid in the supply lumen 93 is both at a higher pressure and a lower temperature than the cooling fluid in the return lumen 95. Advantageously, locating the supply lumen 93 within the return lumen 95 reduces the delivery size of the treatment device 20' and reduces thermal losses in the supply lumen 93. An electrode 90 is applied to an outside surface of the fluid supply channel 97 to form lesions 92 adjacent the bronchus 80 of a patient.

Fluid is circulated by the fluid cooling supply system 12 through the treatment device 20' during energy delivery to the electrode 90. The fluid is circulated serially from the supply lumen 93, through the fluid supply channel 97, into the expandable member 82, and then out the return lumen 95. Fluid circulating through the fluid supply channel 97 and the expandable member 82 protect a region of tissue between an interior wall of an airway and a target treatment region that is located within the airway wall and radially spaced from the interior wall of the airway. In this example, the treatment device 20 uses energy to damage target regions. As used herein, the term "energy" is broadly construed to include, without limitation, thermal energy, cryogenic energy (*e.g.*, cooling energy), electrical energy, acoustic energy (*e.g.*, ultrasonic energy), radio frequency energy, pulsed high voltage energy, mechanical energy, ionizing radiation, optical energy (*e.g.*, light energy), microwave, and combinations thereof, as well as other types of energy suitable for treating tissue. In some embodiments, the treatment device delivers energy and one or more substances (*e.g.*, radioactive seeds, radioactive materials, etc.), treatment agents, and the like. In the example shown in Figures 5 and 5A, the treatment device can include one or more electrodes 90 that are each operable to output ultrasound, microwave, electrical energy, and/or radiofrequency (RF) energy.

In some aspects, fluid is circulated by the fluid cooling supply system 12 directly adjacent the electrode 90. Accordingly, the supply and return lumens may be positioned adjacent the electrode 90, which may provide a high mass flow rate of chilled fluid across a surface of the electrode 90.

5 In another example, an energy delivery portion is located within an expandable member configured to circulate the cooled fluid. For example, an ultrasonic energy delivery device or microwave antennae can be located in an inflatable balloon through which the cooled fluid is circulated.

10 The continuous flow of chilled fluid through the energy delivery device allows the energy delivery portion to form much deeper lesions while delivering the same amount of energy through the tissue of the patient. Thus, treatment is quicker and more effective at the target regions than without providing continuous chilled fluid throughout the treatment device as described in the present disclosure because the nerve tissue at the target regions is more effectively and efficiently damaged.

15 As mentioned above, the heat exchanger discussed with reference to Figures 1-5B could instead be a resilient body, such as a bag, removably coupled to the cooling device with a given biasing force for effectuating heat transfer from fluid contained in or traveling through the bag. The bag may include the same or similar features as the cartridges discussed herein. For example, the bag may have a fluid channel having a serpentine pattern. The bag may have an
20 outlet port in fluid communication with a treatment device positioned in a patient. At least one biasing mechanism may be coupled to the bag and configured to bias the bag with a given force to chill the fluid to a selected temperature for delivery of a patient, such as further described elsewhere in the present disclosure. The at least one biasing mechanism may be a plate removably attached to the cooling device such that the bag is positioned between the cooling
25 device and the plate, or the bag may be biased to the cooling device by other attachment devices, such as with clamps or other devices exhibiting biasing forces to an object. The bag may include a membrane positioned adjacent a cooling device and having a thickness of between 2 millimeters and 4 millimeters, although the thickness may be less than 2 millimeters depending upon the material of the bag. In addition, the bag may be positioned horizontal over a cooling
30 device and the weight, such as a metal plate, may be positioned over the bag to provide a sufficient biasing force to cool the fluid to a desired fluid temperature. The given biasing force between the bag and the cooling device may be between 5 and 10 pounds of force, or may vary beyond such range.

In other embodiments, a cartridge and a bag may be used together. For example, a cartridge may have a slot to receive a bag configured to contain a fluid. The bag may be inserted into the slot and the fluid may be inserted into the bag, thereby inflating the bag in the slot, which provides a sufficient given biasing force between the bag and a thermal surface of the cartridge to effectuate heat transfer of the fluid by a cooling device against which the cartridge is positioned adjacent thereto, for example.

Figures 6A and 6B show a heat exchanger cartridge 28 according to one aspect of the present disclosure. Figure 6A shows the cartridge 28 having a first plate 41 and a second plate 43 secured to each other. The first plate 41 is preferably comprised of a thermally conductive material, such as copper, and includes a thermally conductive surface 98 for biasing to a cooling device 36 (Figures 1 and 2). The first plate 41 may include 0.5 to 1 micron of silver material over the copper material to improve thermal transfer between the fluid in the heat exchanger cartridge 28 and the cooling device 36. This also provides a biocompatible and inert surface for the fluid to contact in the heat exchanger cartridge 28. The second plate 43 is preferably comprised of an insulating material, such as a polymer, ABS, nylon, or polycarbonate. An insulating foam or natural cork insulator could be placed inside the cartridge 28 or on an outer surface of the cartridge 28 to thermally isolate the fluid from the ambient air temperature around the cartridge 28.

The cartridge 28 may have at least one biasing mechanism that may include four magnets 99 secured to the cartridge. The magnets 99 may be secured into bores 100 at respective corners of the second plate 43. Alternatively, one long magnet or a plurality of magnets can be secured along various portions of the cartridge to achieve the same biasing force to a cooling device as further discussed in the present disclosure. Securing the magnets 99 at the four corners of the cartridge 28 provides improved surface-to-surface contact between the thermally conductive surface 98 of the first plate 41 and the thermal plate 38 of the cooling device 36 because the magnets tend to provide uniform biasing force along most or all of the surface area of the thermally conductive surface 98 as biased to the thermal plate 38, thereby improving and maintaining consistent and efficient heat transfer from the fluid during treatments (Figure 7B).

The cartridge 28 further includes an inlet port 102 positioned at an upper portion 104 of a first end 106 of the second plate 43, and an outlet port 108 positioned at a lower portion 110 of a second end 112 of the second plate 43. The inlet port 102 may be coupleable to a fluid reservoir, and the outlet port 108 may be coupleable to a treatment device positioned in a patient.

With continued reference to Figure 6B, the second plate 43 includes a fluid channel 114 in fluid communication with the inlet port 102 and the outlet port 108. The fluid channel 114

serpentine throughout the cartridge in a vertical manner from the upper portion 104 to the lower portion 110 such that any gas in the system may tend to rise toward the upper portion of the fluid channel 114. The fluid channel 114 is formed to have a substantially flat cross sectional area through which fluid is traversed (Figure 6D). This provides one advantage of improving heat transfer from the fluid during treatment because the fluid traverses adjacent the first plate 41 in a substantially thin or flat manner, which can maximize the heat transfer from the fluid by virtue of thermodynamic principles. The second plate 43 further includes a perimeter recess 116 that is formed to receive the first plate 41 such that the thermally conductive surface 98 is substantially flush and planar with a biasing surface 118 of the second plate 43. The perimeter recess 116 may include sealing channels 120 that may receive an adhesive to secure the first plate 41 to the second plate 43 (Figure 6D and 6E). Accordingly, the first plate 41 may be secured to the second plate 43 across various portions of the first plate 41, which can prevent or reduce bulging or distortion of the first plate 41 due to suction forces or other forces. As a result, thermal heat transfer is increased because greater surface-to-surface contact is maintained between the first plate 41 and the thermal plate 38 due to the particular configuration of the cartridge.

Figure 6C shows a heat exchanger cartridge 28' according to one aspect of the present disclosure. The cartridge 28' may include the same or similar features with reference to Figures 6A and 6B. The cartridge 28' includes a first plate 41' and a second plate 43' and four magnets 99 positioned in bores 100 at respective corners of the second plate 43'. Thus, the cartridge 28' includes many of the same or similar features discussed with reference to Figures 6A and 6B with at least one notable difference: the second plate 43' includes a fluid reservoir 122 contained wholly within a cavity 124 of the cartridge 28' such that there is no need for an external fluid reservoir to operate a fluid cooling supply system. From the fluid reservoir 122, a fluid channel 114' serpentine throughout the cartridge in a vertical manner from top to bottom such that any gas in the system may tend to rise to the upper portion of the fluid channel 114' and to the fluid reservoir 122. Providing a fluid reservoir 122 inside the cartridge 28' provides the advantage of improving sterility because it is no longer required to provide an external reservoir with various supply tubes and connections which a practitioner must handle and connect and disconnect between treatments. Providing a fluid reservoir 122 inside the cartridge 28' further provides the advantage of having a disposable cartridge that can be easily manufactured and supplied to practitioners for: quick attachment to a cooling device, sterile use of a fluid during treatment, and easy detachment and replacement of the cartridge between treatments. In some aspects, a small bag may be positioned in the cavity 124 and coupled to the fluid channel 114'. In this manner,

fluid pressure remains constant in the fluid channel 114' during operation because the bag will collapse when fluid is drawn out of the bag.

In some embodiments, corner portions of the fluid channels in each cartridge discussed in the present disclosure may have a relatively large radius, such as illustrated by the shadow lines of a corner portion 121 on Figure 6C. The corner portion 121 provides a gradual transition between horizontal and vertical sidewalls of the fluid channel to help overcome the surface tension of gas bubbles that may otherwise become stuck in the corners of the fluid channel. This will increase the fluid pressure in the cartridge because there will be less gas bubbles in the fluid channel than if the cartridge had corners with a smaller radius, for example.

Figure 6D shows a cross sectional view of the heat exchanger cartridge 28 taken along lines 6D-6D of Figure 6A. Figure 6E show a portion of figure 6D. The features shown in Figures 6D and 6E may include the same or similar features with reference to Figure 6C. The cartridge 28 includes a first plate 41 and a second plate 43 secured to each other. The second plate 43 includes a fluid channel 114 that serpentine through the cartridge 28 adjacent the first plate 43. The second plate 43 includes a perimeter recess 116 and sealing channels 120 that may receive an adhesive to secure the first plate 41 to the second plate 43. Accordingly, the first plate 41 may be secured to the second plate 43 across various portions of the first plate 41, which can prevent or reduce distortion of the first plate 41 due to suction forces or other forces. The configuration shown and discussed with reference to Figures 1-5 allows for a relatively thin first plate 41 (as further discussed herein), which improves heat transfer from the fluid in the fluid channel 114 chilling of the fluid.

The first plate 41 may have a thickness T to maintain a substantially flat surface between the cartridge 28 and the thermal plate 38. If the first plate 41 is too thin for a specified metal, when placed under vacuum, the first plate 41 may exhibit a rippled surface at locations along which the fluid channel 114 is positioned. This may create air pockets between thermally conductive surface 41 and the planar surface 51 of the thermal plate 38, which thereby results in poor heat transfer from the fluid. In some embodiments, the thickness T of the first plate 41 is between .005 inch and .01 inch, but the thickness T may vary beyond such range. Preferably, the thickness T is .01 inch.

In addition, a cross sectional profile of the fluid channel 114 may include a corner R having a radius (Figure 6E), as compared to having a right angle profile (Figure 6D). Corner R is shown at a lower corner portion of the fluid channel for purposes of illustration; corner R is ideally formed at upper portions of the fluid channel to prevent gas bubbles from being trapped in the otherwise right angled corners, particularly near the upper corners where the fluid channel

transitions from one vertical channel section to a horizontal channel section (Figure 6C). Providing rounded corners may increase the fluid pressure in the cartridge because there would be less gas bubbles in the fluid channel than if the channel had corners with a right angle, for example.

5 Figures 7A and 7B show top plan views of a heat exchanger cartridge 28 according to one aspect. The cartridge 28 may have the same or similar features as the cartridges with reference to Figures 1-6E. Accordingly, the cartridge may include a first plate 141 and a second plate 143 secured to each other. The first plate 141 may include a thermally conductive surface 98. Magnets 99 may be secured to the cartridge 28 at opposing ends of the cartridge 28.
10 Likewise, a cooling device 36 having a thermal plate 38 and a hot plate 39 may also have the same or similar features as described with reference to Figures 1-4. The thermal plate 38 includes a planar surface 51 to bias against the thermally conductive surface 98 of the cartridge 29. A spacer 40 may extend around a perimeter of the thermal plate 38 and the hot plate 39 (Figure 2). The spacer 40 may include magnets 53 positioned at corresponding positions relative
15 to the cartridge magnets 99. The spacer 40 may include an outer surface 49 that is substantially planar with the planar surface 51 of the thermal plate 38 to collectively provide a flush surface region over which the cartridge 28 may be biased.

The cartridge 28 may be manufactured or formed to be in a first state A when disengaged from the thermal plate 38 (Figure 7A) and to be in a second state B when engaged to the thermal
20 plate 38 (Figure 7B). Accordingly, Figure 7A shows the cartridge 28 in the first state A (a pre-stressed configuration), which is achieved by forming the first plate 141 and the second plate 143 of the cartridge 28 to have a profile with a convex shape relative to the planar surface 51 of the thermal plate 38. As such, a first end 106 and a second end 112 of the cartridge 28 may be positioned slightly farther away from a central area 115 of the cartridge, which is illustrated by
25 distances X shown on ends 106, 112 of the cartridge 28. As shown in Figure 7B, when the cartridge 28 is engaged to the cooling device 36, because of the pre-stressed shape and magnetic force the cartridge 28 is biased flush to the thermal plate 38. Thus, the cartridge 28 has a profile that is a substantially flat relative to the thermal plate 38 because the cartridge 28 tends to flatten due to magnetic forces. Such configuration and biasing means provide improved surface-to-
30 surface contact between the thermally conductive surface 98 of the first plate 141 and the thermal plate 38 of the cooling device 36, thereby resulting in improved heat transfer while reducing heat losses. Improving heat transfer and reducing heat losses is important during treatment of a patient because some treatment systems, such as the pulmonary treatment system

discussed in the present disclosure, may require a given fluid temperature and a given fluid pressure for a given amount of time during treatment.

Figures 8 and 9 illustrate a treatment system 210 according to one aspect of the present disclosure. Figure 8 shows the treatment system 210 having a fluid cooling supply system 212 and a pulmonary treatment system 217 coupled to each other by a supply line 214 and a return line 216. Figure 9 shows a partially exploded view of certain components of the fluid cooling supply system 12 of Figure 8.

The treatment system 210 shown in Figures 8-13B may have the same or similar features of the systems described and shown with reference to Figures 1-7B. Accordingly, the pulmonary treatment system 217 may include a flexible bronchoscope 18 having a treatment device 20, a control portion 68, a steering mechanism 70, and a video system 72. The flexible bronchoscope 18 may include an insertion tube 74 extending from a control section 76 external to the patient's body, through the trachea 78, and to a treatment site within the left main bronchus 80 of the lungs 81 of a patient. The treatment device 20 can be positioned in the left main bronchus 80, or positioned in other locations, such as within the right main bronchi, the lobar bronchi, and bronchus intermedius. The treatment device 20 can be navigated through tortuous airways to perform a wide range of different procedures, such as, for example, denervation of a portion of a lobe, an entire lobe, multiple lobes, or one lung or both lungs. In some embodiments, the lobar bronchi are treated to denervate lung lobes. Based on the effectiveness of the treatment, the physician can concurrently or sequentially treat additional lobe(s).

The steering mechanism 70 may be coupled to the bronchoscope 18 and may receive the supply line 214 and the return line 216 to allow egress of the lines into the bronchoscope 18 and ultimately to the treatment device 20. The bronchoscope 18 may be coupled to the video system 72, which allows a practitioner to observe progress of the insertion tube 74 through the patient on a monitor 82 as the insertion tube 74 is steered with the assistance of the control portion 68. The video system 72 can also allow a practitioner to determine whether fluid is supplied to the treatment device 20 from the fluid cooling supply system 212. In addition, the bronchoscope 18 may be coupled to the control portion 68 to control some or all aspects of treatment, such as the amount of energy delivered to the treatment device 20. Accordingly, the treatment device 20 of the bronchoscope 18 is in fluid communication with the supply line 214 and the return line 216 of the fluid cooling supply system 212. As such, the fluid cooling supply system 212 is adapted to cool a fluid, pump the fluid, and circulate the fluid through the treatment device 20.

With continued reference to Figures 8 and 9, in some aspects the fluid cooling supply system 212 may include: a housing 232 having a front plate 234; a cooling device 236 having a

thermal plate 238 extending through the front plate 234; a pump 230 for pumping fluid; a heat exchanger cartridge 228 coupled to the front plate 234 and biased to the cooling device 236; a cam system 237 coupled to the front plate 234 for biasing the cartridge 228 to the thermal plate 238; and a controller 242 coupled to the pump 230 and cooling device 236.

5 The housing 232 may include a first portion 231 and second portion 233 secured to each other and to structurally support and house various components of the system. The first portion 231 may include an opening 235 for receiving and supporting a front portion of the cooling device 236. The cooling device 236 includes the thermal plate 238, a hot plate 239, fins 246, and a fan 248, as with commonly available TECs. The thermal plate 238 may include a planar
10 surface 251 for biasing to the cartridge 228. The cooling device 236 may include a support plate 247 secured to the first portion 231 of the housing 232. A spacer 240 may be secured between the cooling device 236 and the front plate 234 for additional support of the cooling device 236 and to allow egress of the thermal plate 238 through the front plate 234.

 In some aspects, the cartridge 228 is slideably coupled to the front plate 234 and biased
15 against the thermal plate 238 of the cooling device 236 (Figures 11A and 11B). As further discussed below, the cartridge 228 may include a fluid reservoir 222 contained within the cartridge 228, or the system may have an external fluid reservoir outside of the cartridge 228 and in fluid communication with the cartridge 228. In the aspect shown, the cartridge includes an outlet port 208 coupled to the supply line 214. The supply line 214 is further coupled along the
20 pump 230 and then through the bronchoscope 18 and to the treatment device 20 in the patient. Accordingly, the supply line 214 is in fluid communication with the treatment device 20, and the return line 216, also in fluid communication with the treatment device 20, extends from the insertion tube 74 and back to the cartridge 228 for recirculation of the fluid during treatment in a closed loop system. Alternatively, the return line 216 may extend to a waste reservoir 219 in an
25 open loop system.

 With continued reference to Figure 9, the front plate 234 is attached to the front portion 231 of the housing 232. The front plate 234 and the housing 232 cooperate to structurally support the cooling device 236 and the pump 230. The front plate 234 includes an opening 243 for receiving the thermal plate 238 of the cooling device 236 and for facilitating biasing of the
30 cartridge 228 to the cooling device 236. The front plate 234 may include an opening 244 for receiving a portion of the pump 230. The pump 230 may include a cover 250 and a rotating device 259 for coupling to the supply line 214. Importantly, the pump 230 is positioned downstream of the cartridge 228 such that the fluid in the cartridge 228 experiences a negative fluid pressure and such that the fluid supplied to the treatment device 20 experiences a positive

fluid pressure during normal operation of the treatment system. The front plate 234 may include control devices 262 coupled to the controller 242 for controlling aspects of the system. The controller 242 may be coupled to the pump 230 for regulating the speed and direction of the pump 230, thereby regulating the direction of flow and volume of fluid circulating through the system. The controller 242 may also be coupled to the cooling device 236 to regulate the temperature of the fluid in the cartridge 228, thereby further regulating the temperature of the fluid circulating through the treatment device 20, and thereby regulating the temperature of patient tissue during treatment. It will be appreciated that the treatment device 20 discussed with reference to Figures 8 and 9 may include the same or similar features discussed with reference to Figures 1-7B, and particularly with reference to Figure 5.

Figure 10 shows a schematic of a treatment system 310 according to one aspect, which may include some or all of the features of Figures 8 and 9. The treatment system 310 includes a fluid cooling supply system 212 coupled to a treatment device 20 positioned in a patient 264. The fluid cooling supply system 212 includes a cooling device 236, a heat exchanger 228, a pump 230, and a controller 242. The controller 242 may be coupled to the cooling device 236 and the pump 230 to regulate temperature and fluid circulation. The heat exchanger 228 may be removably coupled to the cooling device 236. A supply path 266 originates at a fluid reservoir 222 contained wholly within the heat exchanger 228. The supply path 266 extends through the heat exchanger 228 and through the pump 230 and terminates at the treatment device 20 for supplying chilled fluid to the patient 264. The return path 268 originates at the treatment device 20 and may return either to the fluid reservoir 222 for recirculation or to a waste reservoir 219. Accordingly, the fluid may be drawn from the reservoir 222 through the heat exchanger 228 at a negative pressure by the pump 230. The fluid is chilled by the cooling device 236 as it travels through the heat exchanger 228. The fluid is supplied to the treatment device 20 at a positive pressure by the pump 230. The fluid may then be circulated through the treatment device 20 and returned from the treatment device 20 to outside the patient 264.

The pump 230 may include forward and reverse gears, as depicted by arrows P, to draw and push the fluid forward through the heat exchanger 228 during treatment. The forward gear draws fluid from the heat exchanger 228 during normal operation of the system 310. Conversely, the reverse gear may push the fluid in reverse through the heat exchanger 228 to expel gas that may exist in the system 310. The speed and direction of the pump 230 may be controlled by the controller 242.

In some aspects, the pump 230 is coupled to a controller for variable control over the speed of the pump in order to control the amount of fluid delivery to the treatment device. Thus,

the size and apposition pressure of the treatment device may be controlled by the variable speed controller. Moreover, a non-contact pressure measurement device may be electrically coupled to the pump and positioned proximate the high pressure side of the fluid path to regulate system pressure, such as by varying the speed of the pump in response to the pressure measured by the non-contact pressure measurement device, for example.

Figures 11A-13C show certain aspects of the front plate 234, the cam system 237, and the cartridge 228 of a fluid cooling supply system 212. Figures 11A and 11B show a back perspective view of the front plate 234 and cartridge 228. The front plate may include a cam system 237 that, when actuated between an engaged state E and a disengaged state D, allows for removal of the cartridge 228. Figure 12A shows a side elevational view of the front plate 234 and Figure 12B shows a cross sectional view of the front plate 234, the cam system 237, the cartridge 228, and the cooling device 236, taken along lines 12B-12B of Figure 11B. Figures 13A-13C show various views of the cartridge 228.

With continued reference to Figures 11A and 11B, the cartridge 228 includes a first plate 241 and a second plate 243 secured to each other. The first plate 241 includes a thermally conductive surface 298 for biasing to a thermal plate of a cooling device (Figures 9 and 12B). The front plate 234 includes an opening 243 and a receiving surface 245. The opening 243 may be sized to facilitate biasing of the cartridge 228 to the thermal plate of a cooling device 236. The receiving surface 245 is sized to receive a portion of the cooling device 236 such that the thermal plate 238 may extend partially through the opening 243. The front plate 234 may also have an opening 244 to receive a pump for pumping fluid through the cartridge 228. The front plate 234 may contain and support the cam system 237 for biasing the cartridge 228 to the cooling device. In some configurations, the cam system 237 includes a cam lever 338 coupled to a cam shaft 340 having four cam lobes 342. The cam lever 338 may be directly attached to the cam shaft 340, or it may be dynamically linked to the cam shaft 340 in other configurations. The four cam lobes 342 are formed along a length of the cam shaft 340 and spatially separated from each other. The cam system 237 may include an actuation member 344 and actuation devices 346. Each actuation device 346 may be comprised of a piston rod 354 and a spring 356 positioned below respective piston rods 354. The actuation devices 346 may be at least partially positioned in respective bores 348 of the actuation member 344 and may be positioned adjacent respective cam lobes 342 (Figure 12B) such that rotation of the cam lobes 342 actuates the pistons 354 in a downward direction.

When the cam system 237 is in the disengaged state D, the cam system 237 is positioned to allow the front plate 234 to slideably receive the cartridge 228. Once the cartridge 228 fully

engaged into the front plate 234, the cam system 237 may be actuated to the engaged state E by rotating the cam lever 338 and cam shaft 340 in a downward rotational direction depicted by arrow C in order to secure the cartridge 228 in the front plate 234 and to bias cartridge 228 to the thermal plate 38 of the cooling device 236 (Figure 12B). Thus, when moved to the engaged state E, the cam lobes 342 simultaneously bias against respective piston rods 354 of the actuation devices 346, which tends to force the actuation devices 346 downwardly in a direction depicted by arrow F, which tends to force the actuation member 344 against the cartridge 228 approximately in a direction depicted by arrow G, which is further discussed below (Figure 12B). Conversely, when the cam system 237 is moved from the engaged state E to the disengaged state D for removal of the cartridge 228, by virtue of actuating the cam lever 338 in a direction depicted by arrow B, the cam shaft 340 and cam lobes 342 rotate in a similar direction, which tends to remove the force applied to the actuation devices 346, which tends to remove the force applied by the actuation member 344 so that the cartridge 228 may be removed (Figure 11A). As previously discussed, providing a given and sufficient biasing force between the cartridge and a cooling device improves surface-to-surface contact between the cartridge and the cooling device, which aids in the effective and efficient cooling of fluid traversing through the cartridge for supply to a patient.

Figure 12A shows a left side elevation view of the front plate 234 of Figure 11A according to one aspect of the present disclosure. The front plate 234 including a slot 358 sized to loosely receive a cartridge 228 when the cam system 237 is in the disengaged state D. The front plate 234 includes an upper biasing surface 360 and a lower biasing surface 362 sized to closely receive the cartridge 228. The upper biasing surface 360 is formed at an angle that is substantially non-parallel to the planar surface 251 of the thermal plate 238 (Figure 12B). Likewise, the lower biasing surface 362 is formed at an angle that is substantially non-parallel to the planar surface 251 of the thermal plate 238. Thus, the slot 358 may have a trapezoid-shaped cross sectional profile to receive the cartridge 228, and the cartridge 228 may also have a corresponding trapezoid-shaped cross sectional profile (Figure 13C). Figure 12A further shows the cam lever 338 in the disengaged state D and a recess portion 364 for allowing passage of a supply line 214 (Figures 8 and 9).

Figure 12B shows a cross sectional view of the front plate 234, the cartridge 228 positioned in the front plate 234, the cooling device 236 and thermal plate 238 positioned adjacent the cartridge 228, and the cam system 237 in the engaged state E. Regarding the cam system 237, the actuation member 344 includes a lower actuation surface 366 that is formed at an angle relative to the planar surface 251 of the thermal plate 238. As further discussed above,

when engaging the cam system 237 via the cam lever 338 and the cam shaft 340, the cam lobes 342 force the actuation devices 346 downwardly and, therefore, the actuation devices 346 force the actuation member 344 downwardly in a direction depicted by arrow F. Consequently, the lower actuation surface 366 biases an upper angular surface 368 of the cartridge 228 and, simultaneously, the lower biasing surface 362 tends to bias a lower angular surface 374 of the cartridge 228, which tends to force the cartridge 228 inwardly in a direction depicted by arrow G. This configuration and operation results in the cartridge 228 biased in approximately a lateral direction against the thermal plate 238 with a given force to effectuate heat transfer from the fluid and to improve surface-to-surface contact between the cartridge 228 and the cooling device 236. This is accomplished, in part, because of the trapezoid-shaped profiles of the slot 358 and the cartridge 228 and because of the angular surface of the actuation member 344, which collectively tend to "slide" the cartridge 228 along respective angled surfaces and into position in the direction depicted by arrow G. Thus, the cam system 237, the front plate 234, and the cartridge 228 are sized to cooperatively operate to bias the cartridge 228 to the thermal plate 238 to effectuate heat transfer from the fluid contained in the cartridge 228.

Figure 13A shows a front perspective view of a cartridge 228 according to one aspect of the present disclosure. The cartridge 228 includes a handle 370 positioned at a left end of the cartridge for easy insertion and removal of the cartridge 228 into the slot 358 of the front plate 234, as previously discussed. The cartridge 228 includes an upper angular surface 368 and a lower angular surface 374 that are formed at respective angles to permit insertion of the cartridge 228 into the front plate 234. The cartridge 228 further includes a plurality of cavities 372 defined by a plurality of cross members 375. The cavities 372 are sized and formed along the front portion cartridge to improve heat transfer from the fluid in the cartridge 228 during operation of the system.

Figure 13B shows a back perspective view of a cartridge 228 according to one aspect of the present disclosure. The cartridge 228 includes a first plate 241 and a second plate 243 attached to each other. The first plate 241 is preferably comprised of a copper material and includes a thermally conductive surface 298 for biasing to a cooling device (Figure 12B). The first plate 241 may include 0.5 to 1 micron of silver material over the copper material to improve thermal transfer between the fluid in the heat exchanger cartridge 228 and the cooling device 236. This also provides a biocompatible and inert surface for the fluid to contact in the heat exchanger cartridge 228. The second plate 43 is preferably comprised of an insulating material, such as ABS, nylon, or polycarbonate. An insulating foam or natural cork insulator could be placed inside the cartridge 28 or on an outer surface of the cartridge 28 to thermally isolate the

fluid from the ambient air temperature around the cartridge 28. The second plate 243 includes a fluid reservoir 322 positioned at an upper portion 376 of the cartridge 228. A fluid channel 314 is formed on the second plate 234 and is in fluid communication with the fluid reservoir 322. The fluid channel 314 serpentine throughout the cartridge in a vertical manner from top to bottom such that any gas in the system tends to rise to the upper portion 376 of the fluid channel 314 and into the fluid reservoir 322. The second plate 243 may include a sealing surface 240 that is recessed to receive the first plate 241. The sealing surface 240 may receive an adhesive to secure the first plate 241 to the second plate 243. Thus, the first plate 241 is secured to the second plate 243 across various portions of the first plate 241, which prevents distortion of the copper plate due to suction forces or other forces acting on the first plate 241.

The second plate 243 may include an outlet port 308 positioned at a lower portion 378 of the cartridge 228 and in fluid communication with the fluid channel 314 and the fluid reservoir 322. The outlet port 308 may be coupled to a supply line for supplying fluid to a patient. In some aspects, the cartridge 228 may include an inlet port 302 in fluid communication with the fluid reservoir 322. The outlet portion 302 may be coupleable to a return line for returning fluid from within the patient. In some aspects, the fluid reservoir 322 may contain a collapsible bag in fluid communication with the fluid channel 314 and the outlet port 308 so that fluid pressure forces experienced by the system are reduced or minimized. In some aspects, the cartridge 228 may not have a fluid reservoir 322 contained in the cartridge; it may simply have a fluid channel coupled to an external reservoir, such as shown in Figure 1.

Figure 13C shows a cross sectional view of the cartridge 228 of Figure 13A and 13B taken along lines 13C-13C. The cartridge 228 includes a first plate 241 and a second plate 243 attached to each other. The first plate 243 includes a thermally conductive surface 298 positioned adjacent a fluid reservoir 322 and a fluid channel 314. The second plate 243 includes the fluid reservoir 322 positioned at an upper portion 376 of the cartridge 228 and the fluid channel 314 in fluid communication with the fluid reservoir 322. The cartridge 228 may include a plurality of cavities 372 defined by a plurality of cross members 375 (Figure 13A). The cavities 372 are sized and formed along the front portion cartridge to reduce the average thickness of the second plate, which consequently will improve heat transfer from the fluid in the cartridge 228 during operation of the cooling system. The cartridge 228 includes an upper angular surface 368 and a lower angular surface 374. The cartridge 228 has a profile to allow insertion of the cartridge 228 into the slot 358 of the front plate 234 for biasing to the cooling machine 236, as further discussed above.

The first plate 241 may have a thickness T to maintain a flat surface between the cartridge 228 and the thermal plate 238. If the first plate 241 is too thin for a specified metal, when placed under vacuum, the first plate 241 may exhibit a rippled surface at locations along which the fluid channel 314 is positioned. This may create air pockets between thermally
5 conductive surface 251 and the thermal plate 238, which thereby results in poor heat transfer from the fluid. In some embodiments, the thickness T of the first plate 241 is between .005 inch and .01 inch, but the thickness T may vary beyond such range. Preferably, the thickness T is .01 inch.

As discussed above with reference to Figures 1-5A, the heat exchanger cartridge
10 discussed with reference to Figures 8-13C could instead be a resilient body, such as a bag, removably coupled to the cooling device with a given biasing force for effectuating heat transfer from fluid contained in or traveling through the bag. The bag may include the same or similar features as the cartridges discussed in the present disclosure. For example, the bag may have a fluid channel having a serpentine pattern. The bag may have an outlet port in fluid
15 communication with a treatment device positioned in a patient. At least one biasing mechanism may be coupled to the cooling device configured to bias the bag with a given force to chill the fluid to a selected temperature for delivery of a patient, such as further described elsewhere in the present disclosure. The at least one biasing mechanism may be the cam system 237 described above. Accordingly, a bag having a fluid chamber for holding a fluid may be inserted into a slot
20 and a biasing member, such as a plate, may be actuated by the cam system to bias the biasing member against the bag, thereby biasing the bag against the cooling device with a given biasing force. Thus, the bag may be replaceable with another bag by disengaging the cam system and the biasing plate from the bag to allow removal of the bag, similar or the same as described with reference to Figures 8-13C.

Figures 14-20 illustrate a system 410 that includes a fluid cooling supply system 412
25 coupled to a treatment system 417. The treatment system 410 shown in Figures 14-20 may have the same or similar features of the systems described and shown with reference to Figures 1-7B and 8-13C.

In the example in Figure 14, the fluid cooling supply system 412 is coupled to the
30 treatment system 417. As described previously, the treatment system 417 may be positionable at least partially in a patient 464 (Figure 15). The fluid cooling supply system 412 is configured to chill a fluid, pump the fluid, and supply the fluid through the treatment system 417. In a closed loop system, the fluid cooling supply system 412 may include a fluid reservoir 422, a fluid 424, a cooling system 426, a heat exchanger cartridge 428, a supply line 414 and a return line 416,

which collectively cooperate to circulate cooled fluid through the treatment system 417 during treatment. The supply line 414 originates at the fluid reservoir 422 and extends through the cartridge 428 and along a pump 430. The supply line 414 may extend through a pulse damper (not depicted) for damping vibration of the supply line 414 during operation of the pump 430.

5 Finally, the supply line 414 extends into the treatment system 417 positionable in a patient. The return line 416, in fluid communication with the supply line 414, originates at the treatment system 417 and extends from inside the patient and back to the reservoir 422 for recirculation of the fluid during treatment. In some embodiments, supply line 414 and return line 416 are connected to fluid reservoir 422 by a coaxial double spike 423 (Figures 15-16), which will be
10 discussed in more detail below.

With reference to Figure 15, and similar to the description of previous embodiments, in some aspects the pulmonary treatment system 417 may include a flexible bronchoscope 418 having a treatment device 420, a control portion 468, a steering mechanism 470, and a video system 472. The flexible bronchoscope 418 may include an insertion tube 474 extending from a
15 control section 476 external to the patient's body, through the trachea 478, and to a treatment site within the left main bronchus 480 of the lungs 481 of a patient. The treatment device 420 can be positioned in the left main bronchus 480, or positioned in other locations, such as within the right main bronchi, the lobar bronchi, and bronchus intermedius. The treatment device 420 can be navigated through tortuous airways to perform a wide range of different procedures, such as, for
20 example, denervation of a portion of a lobe, an entire lobe, multiple lobes, or one lung or both lungs. In some embodiments, the lobar bronchi are treated to denervate lung lobes. Based on the effectiveness of the treatment, the physician can concurrently or sequentially treat additional lobe(s).

The steering mechanism 470 may be coupled to the bronchoscope 418 and may receive
25 the supply line 414 and the return line 416 to allow egress of the lines into the bronchoscope 418 and ultimately to the treatment device 420. The bronchoscope 418 may be coupled to the video system 472, which allows a practitioner to observe progress of the insertion tube 474 through the patient on a monitor 482 as the insertion tube 474 is steered with the assistance of the control portion 468. The video system 472 can also allow a practitioner to determine whether fluid is
30 supplied to the treatment device 420 from the fluid cooling supply system 412. In addition, the bronchoscope 418 may be coupled to the control portion 468 to control some or all aspects of treatment, such as the amount of energy delivered to the treatment device 420. Accordingly, the treatment device 420 of the bronchoscope 418 is in fluid communication with the supply line 414 and the return line 416 of the fluid cooling supply system 412. As such, the fluid cooling supply

system 412 is adapted to cool a fluid, pump the fluid, and circulate the fluid through the treatment device 420.

5 With reference to Figure 16, which shows a partially exploded view of certain components of the fluid cooling supply system 412 of Figures 14-15, and similar to the description of previous embodiments, in some aspects the fluid cooling supply system 412 may include: a housing 432 having a front plate 434; a cooling device 436 having a thermal plate 438
10 extending through the front plate 434; a pump 430 for pumping fluid; a heat exchanger cartridge 428 removably coupled to the front plate 434 and biased to and into contact with the cooling device 436; a hinged door 437 hingedly coupled to the front plate 434 of housing 423 for biasing the cartridge 428 to the thermal plate 438; and a controller 442 (Figure 18) coupled to the pump 430 and cooling device 436.

The housing 432 may include a first portion 431 and second portion 433 secured to each other and to structurally support and house various components of the system. The first portion 431 may include an opening 435 for receiving and supporting a front portion of the cooling
15 device 436. The cooling device 436 includes the thermal plate 438, a hot plate 439, fins 446, and a fan 448, as with commonly available TECs. The thermal plate 438 may include a planar surface 451 for biasing to the cartridge 428. The cooling device 436 may include a support plate 447 secured to the first portion 431 of the housing 432.

The front plate 434 may include an opening 444 for receiving a portion of the pump 430.
20 The pump 430 may include a cover 450 and a rotating device 459 for coupling to the supply line 414. Pump 430 is positioned downstream of the cartridge 428 such that the fluid in the cartridge 428 experiences a negative fluid pressure and such that the fluid supplied to the treatment device 420 experiences a positive fluid pressure during normal operation of the treatment system.

With continued reference to Figure 16, the front plate 434 may be attached to the front
25 portion 431 of the housing 432. The front plate 434 and the housing 432 cooperate to structurally support the cooling device 436 and the pump 430. The front plate 434 includes an opening 443 for receiving the thermal plate 438 of the cooling device 436. In some embodiments, hinged door 437 pivots relative to the front plate 434 between an open position and a closed position. In the open position, cartridge 428 can be inserted into or removed from
30 opening 443. In the closed position, cartridge 428 is biased against the thermal plate 438 of the cooling device 436.

Figure 17 shows back perspective view of a hinged door assembly 440 according to one aspect of the present disclosure. In some embodiments, the hinged door 437 can have at least one biasing mechanism that may include a plurality of magnets 453 (Figure 17) for securing the

hinged door 437 in the closed position. The magnets 453 may be secured into bores 455 defined in hinged door 437. Alternatively, one long magnet or a plurality of magnets can be secured along various portions of hinged door 437 or front plate 434 to achieve the same biasing force as discussed in the present disclosure. The biasing force provides improved surface-to-surface contact between the heat exchanger cartridge 428 and the thermal plate 438 of the cooling device 436, thereby improving and maintaining consistent and efficient heat transfer from the fluid during treatments.

Hinged door assembly 440 can also include a plurality of hinges 460 to couple the hinged door 437 to the front plate 434 or first portion 431 of housing 432. In one embodiment, the hinged door assembly 437 can include two hinges 460, thereby allowing hinged door 437 to pivotably shift relative to front plate 434 or first portion 431 of housing 432 between an open position and a closed position.

In one embodiment, hinged door 437 can be defined by a cutout 461 sized to receive and house a portion of heat exchanger cartridge 428. In one embodiment, hinged door 437 can include one or more cutouts 462 sized to receive and house the inlet port 402 and outlet port 408 and associated bubble traps of heat exchanger cartridge 428.

Figure 18 shows a schematic of a treatment system 410 in accordance with one embodiment of the invention, and similar to the schematic of treatment system 310 of Figure 5. The treatment system 410 includes a fluid cooling supply system 412 coupled to a treatment device 420 positioned in a patient 464. In an embodiment, and similar to Figure 5, the treatment device 420 can include one or more electrodes 90 that are each operable to output ultrasound, microwave, electrical energy, and/or radiofrequency (RF) energy.

The fluid cooling supply system 412 includes a cooling device 436, a heat exchanger 428, a pump 430, and a controller 442. The controller 442 may be coupled to the cooling device 436 and the pump 430 to regulate temperature and fluid circulation. The heat exchanger 428 may be removably coupled to the cooling device 436. A fluid supply line 414 originates at a fluid reservoir 422. The fluid supply line 414 continues to the heat exchanger 428 and through the pump 430 and terminates at the treatment device 420 for supplying chilled fluid to the patient 464. A fluid return line 416 continues from the treatment device 420 and may return either to the fluid reservoir 422 for recirculation and/or to a waste reservoir 419. Accordingly, the fluid may be drawn from the reservoir 422 through the heat exchanger 428 at a negative pressure by the pump 430. The fluid is chilled by the cooling device 436 as it travels through the heat exchanger 428. The fluid is supplied to the treatment device 420 at a positive pressure by the pump 430.

The fluid may then be circulated through the treatment device 420 and returned from the treatment device 420 to outside the patient 464.

A control portion 468 (Figure 15) can be coupled to the controller 442 for controlling aspects of the system. The controller 442 may be coupled to the pump 430 for regulating the speed and direction of the pump 430, thereby regulating the direction of flow and volume of fluid circulating through the system. The controller 442 may also be coupled to the cooling device 436 to regulate the temperature of the fluid in the cartridge 428, thereby further regulating the temperature of the fluid circulating through the treatment device 420, and thereby regulating the temperature of patient tissue during treatment.

The pump 430 may include forward and reverse gears, as depicted by arrows P, to draw and push the fluid forward through the heat exchanger 428 during treatment. The forward gear draws fluid from the heat exchanger 428 during normal operation of the system 410. Conversely, the reverse gear may push the fluid in reverse through the heat exchanger 428 to expel gas that may exist in the system 410. The speed and direction of the pump 430 may be controlled by the controller 442.

In some aspects, the pump 430 is coupled to a controller for variable control over the speed of the pump in order to control the amount of fluid delivery to the treatment device. Thus, the size and apposition pressure of the treatment device may be controlled by the variable speed controller. Moreover, a non-contact pressure measurement device may be electrically coupled to the pump and positioned proximate the high pressure side of the fluid path to regulate system pressure, such as by varying the speed of the pump in response to the pressure measured by the non-contact pressure measurement device, for example.

Figures 19A and 19B shows a front perspective view and a back perspective view, respectively, of a thermoformed heat exchanger cartridge 428 according to one aspect of the present disclosure. The heat exchanger cartridge 428 includes a flexible thermoformed tray 443 coupled or bonded to a first plate 441. The tray 443 coupled to the plate 441 defines a fluid channel 450 for fluidly coupling an inlet supply line 414a (uncooled fluid) and an outlet supply line 414b (cooled fluid) for introduction to the treatment device.

The first plate 441 is preferably comprised of a copper material and includes a thermally conductive surface 498 for biasing to the planar surface 451 of cooling device 436 (Figure 16). For example, the first plate 441 can comprise a conductive material deposited on, such as by plating, coating (e.g. conductive inks or coatings), and/or laminating (e.g. thin films), a copper material to improve optimize thermal transfer between the fluid in the heat exchanger cartridge 428 and the cooling device 436. The conductive material can comprise silver, parylene,

aluminum, or combinations thereof. The first plate 441 may have a thickness T to maintain a flat surface between the cartridge 428 and the thermal plate 438. If the first plate 441 is too thin for a specified metal, when placed under vacuum, such as when the pump is in reverse, the first plate 441 may exhibit a rippled surface at locations along which the fluid channel 450 is positioned.

5 This may create air pockets between the first plate 441 and the thermal plate 438, which thereby results in poor contact, and therefore poor heat transfer from the fluid. It can also cause complete collapse of the recess of the tray thereby occluding the fluid channel. In some embodiments, the thickness T of the first plate 441 is between .005 inch and .015 inch, but the thickness T may vary beyond such range. Preferably, the thickness T is about .010 inch (or 10

10 mil).

In one particular embodiment, the plate 441 comprises a copper plate having a thickness of about 10 mil coated with a 0.5 to 1 micron of silver material. This provides a biocompatible and inert surface for the fluid to contact in the heat exchanger cartridge 428. Optionally, a parylene coating is provided over at least a portion of the silver material to provide barrier

15 properties and to aid in sealing or bonding the tray 443 to the plate 441.

The thermoformed tray 443 is preferably comprised of transparent or translucent thermoformed material, such as polyvinyl chloride (PVC) or polyethylene terephthalate (PET). The material of the tray 443 exhibits sufficient flexibility such that it remains adhered to the plate 441 under high temperature applications such as sterilization. A thickness of the tray 443 is

20 optimized to provide sufficient rigidity to the tray 443 such that when the system is pressurized in reverse, such that a vacuum is pulled, the tray does not deform and ripple so as not to compromise or significantly diminish heat exchanging characteristics of the cartridge 428. Optionally, an insulating foam or natural cork insulator (not shown) can be placed inside the cartridge 428 or on an outer surface of the cartridge 428 to thermally isolate the fluid from the

25 ambient air temperature around the cartridge 428.

The tray 443 further includes a rim or recess 452 spaced inwardly from a perimeter of tray 443 by a sealing surface or flange 440. The sealing surface 440 may receive an adhesive, such as a UV activated or curable adhesive or an epoxy, to secure the plate 441 and tray 443 together. Any excess adhesive is collected into the recess 452 such that it does not interfere with

30 fluid channel 450. Thus, the first plate 441 is secured to the tray 443 across various perimeter portions of the first plate 441, and prevents distortion of the copper plate due to suction forces or other forces acting on the first plate 441. In one embodiment, the first plate 441 and tray 443 are bonded using a UV curable adhesive and exposing the assembly to UV radiation. The

transparency of the tray material allows for sufficient exposure to UV radiation to adequately cure the adhesive.

The fluid channel 450 is defined as the spaced between a recessed area formed in tray 443 and the first plate 441. The fluid channel acts to provide fluid communication between the fluid reservoir 422 and the pump 430 and ultimately the treatment device 420, while cooling the fluid passing through the fluid channel 450. In an embodiment, the fluid channel 450 serpentine throughout the cartridge 428 for a desired number of passes, such as seven passes as depicted. The number of passes as well as the depth and width of the channel 450 is selected based on a desired residence time of the fluid within the cartridge 428 to cool the fluid to a desired temperature. For example, the fluid reservoir 450 is dimensioned to provide adequate residence time for a coolant such as saline to be cooled from room temperature (25°C) to about 0.1°C-10°C, and more particularly to about 1°C -6°C, and even more particularly, to about 3°C-5°C, at a flow rate of about 100 mL/min.

Referring to Figure 19A, an inlet port 402 is formed at a second end 450a of the fluid channel 450, and is in fluid communication with the inlet supply line 414a, which is in fluid communication with the fluid reservoir 422. An outlet port 408 is formed at a second end 450b of the fluid channel 450, is in fluid communication with the outlet supply line 414b. This portion of the supply line 414 is in fluid communication with the pump 430 and then through the bronchoscope 418 and to the treatment device 420 in the patient. In some aspects, the inlet port 402 and outlet port 408 can have a sloped profile with a starting depth greater than the depth of the fluid channel 450, sloping downward to a depth equal to the depth of the fluid channel. This allows for pooling of inlet fluid with a head space for the collection of gas bubbles in the system. In other words, any gas in the system tends to rise to the head space of ports 402 and 408.

The heat exchanger cartridge 428 has a profile to allow insertion of the cartridge 428 into a space between hinged door 437, and thermal plate 438 of cooling device 436. For example, in use, the cartridge 428 can be inserted into opening 443 when the hinged door 437 is open. When the hinged door 437 is closed, the cartridge 428 is biased against the thermal plate 438 of the cooling device 436, thereby sandwiching cartridge 428 between hinged door 437 and the thermal plate 438 of the cooling device 436 as further discussed above. In some embodiments, cartridge 428 can further be flanked by opening 443 defined in front plate 434. In some embodiments, heat exchanger cartridge 428 can include one or more notches 479 defined in one or more edges of the cartridge 428. The notches 479 can be sized to receive keys 481 of the front plate 434 for the purpose of ensure that cartridge 428 is inserted in an orientation that allows for normal operation.

Referring now to Figure 20, a perspective view of a coaxial bag spike assembly 423 for coupling the fluid reservoir 422 to the system 410 according to one aspect of the present disclosure is depicted. In some embodiments, supply line 414 and return line 416 can be connected to fluid reservoir 422 in a single location via the coaxial bag spike assembly 423 (*see, for also*, Figures 15-16). In this embodiment, return line 416 is in fluid connection with an inner channel 425, and supply line 414 is in fluid connection with an outer channel 427, coaxial with the inner channel 425. Both the inner and outer channels 425 and 427 are in fluid connection with the fluid reservoir 422 and the fluid 424 there within, but are isolated from each other. However, the reverse configuration can also be contemplated.

In one embodiment, the assembly 423 comprises a hypotube 429 having an internal diameter 431 inserted through a lumen of an injection molded non-vented spike female luer 500 having an internal diameter 502 greater than the internal diameter 431 of the hypotube 429 to create coaxial outer channel 427 and inner channel 425. The assembly 423 can further include an injection molded vented spike cap 504 coupled to a first end of the female luer 500, to fluidly connect the fluid reservoir 422 with the outer and inner channels 425. The assembly 423 also includes an injection molded bag spike adapter 506 having a first port 508 for coupling inlet supply line 414 to outer channel 427, and a second port 510 for coupling outlet supply line 316 to inner channel 425 of hypotube 429. The reverse configuration (i.e. coupling inlet supply line 414 to the second port 510 and outlet supply line 416 to first port 508) can also be contemplated.

The internal diameter 431 of the hypotube is preferably sized to control back pressure in the treatment system 417, and or to effect pressure in an expandable member of the treatment device. Optionally, various clamps (not shown) can be used anywhere along the supply line 414 and/or return line 416 to further regulate supply and/or return flow of fluid to and from the reservoir 422. Accordingly, coaxial double spike 423 precludes the need for a separate supply and return spike by allowing fluid 424 to flow out of and into fluid reservoir simultaneously and at the same location on the reservoir 422.

The various embodiments and aspects described above can be combined to provide further embodiments and aspects. These and other changes can be made to the embodiments in light of the above-detailed description. The aspects, embodiments, features, systems, devices, materials, methods and techniques described herein may, in some embodiments, be similar to any one or more of the embodiments, features, systems, devices, materials, methods and techniques described in U.S. Patent No. 8,088,127, PCT Application No. PCT/US2010/056424 filed November 11, 2010 (Publication No. WO 2011/060200), U.S. Application No. 12/913,702 filed on October 27, 2010, U.S. Application No. 12/944,666 filed November 11, 2010, U.S.

Application No. 13/081,406 filed on April 6, 2011, and U.S. Provisional Application No. 61/543,759. Each of these applications is incorporated herein by reference in its entirety. In addition, the aspects, embodiments, features, systems, devices, materials, methods and techniques described herein may, in certain embodiments, be applied to or used in connection
5 with any one or more of the embodiments, features, systems, devices, materials, methods and techniques disclosed in the above-mentioned applications and patents.

Unless the context requires otherwise, throughout the specification and claims which follow, the word “comprise” and variations thereof, such as “comprises” and “comprising” are to be construed in an open, inclusive sense, that is, as “including but not limited to.”

10 In general, in the following claims, the terms used should not be construed to limit the claims to the specific embodiments and aspects disclosed in the specification and the claims, but should be construed to include all possible embodiments and aspects along with the full scope of equivalents to which such claims are entitled. Accordingly, the claims are not limited by the disclosure.

15

CLAIMS

1. A system for treatment of a patient, comprising:
a fluid cooling supply device configured to draw a fluid through a heat exchanger at a negative pressure to chill the fluid and to deliver the chilled fluid to the patient at a positive pressure; and
an energy delivery device positioned in the patient, the energy delivery device coupled to the fluid cooling supply device such that the fluid cooling supply device circulates the chilled fluid through the energy delivery device to cool the energy delivery device during treatment of the patient.
2. The system of claim 1, the fluid cooling supply device further comprising a pump positioned in a supply path between the heat exchanger and the energy delivery device, the pump configured to draw the fluid through the heat exchanger and configured to circulate the fluid through the energy delivery device.
3. The system of claim 1, further comprising a heat exchanger cartridge coupled to the fluid cooling supply device, the heat exchanger cartridge having a thermally conductive surface and a fluid channel that extends through the cartridge with at least a portion of the fluid channel arranged adjacent to the thermally conductive surface, wherein the fluid channel is in fluid communication with the energy delivery device.
4. The system of claim 1, wherein the temperature of the fluid delivered by the fluid cooling supply device is such that the temperature at the energy delivery device is maintained at or below 20°C during treatment of the patient.
5. The system of claim 1, wherein a temperature of the fluid delivered by the fluid cooling supply device is such that the temperature at the energy delivery device is maintained between 20°C and -5°C during treatment of the patient.
6. The system of claim 1, wherein a temperature of the fluid delivered by the fluid cooling supply device is such that the temperature at the energy delivery device is maintained between 5°C and -2°C during treatment of the patient.

7. The system of claim 1, wherein a temperature of the fluid delivered by the fluid cooling supply device is such that the temperature at the energy delivery device is maintained between 20°C and -5°C, and wherein the temperature at the energy delivery device is maintained for a selected amount of time during a treatment portion of treatment of the patient.

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8. The system of claim 7, wherein the selected amount of time for the treatment portion is less than 60 seconds.

9. The system of claim 7, wherein the selected amount of time for the treatment portion is
10 between 60 seconds and 120 seconds.

10. The system of claim 1, wherein the energy delivery device includes an electrode adapted to deliver energy to a target tissue of the patient, and wherein the energy delivery device comprises at least one lumen to allow circulation of the fluid through the energy delivery device
15 from the fluid cooling supply device to cool the electrode during treatment.

11. The system of claim 1, wherein the energy delivery device includes an electrode coupled to a cooling member, the electrode and the cooling member arranged adjacent to a wall of an airway of the patient such that delivery of energy to the electrode and circulation of chilled fluid
20 through the cooling member damages nerve tissue so that nervous system signals in the patient are attenuated.

12. The system of claim 1, wherein the energy delivery device is configured to deliver the chilled fluid to the patient a fluid flow rate of between 70 milliliters to 160 milliliters per minute.

25

13. The system of claim 1, the fluid cooling supply device further comprising a pump positioned in a return path downstream of the energy delivery device, the pump configured to draw the fluid through the heat exchanger and the energy delivery device.

30 14. The system of claim 1, the fluid cooling supply device further comprising a pump positioned in a supply path between the heat exchanger and the energy delivery device, the pump configured to draw the fluid through the heat exchanger and configured to circulate the fluid through the energy delivery device, and a supplemental pump positioned in a return path

downstream of the energy delivery device and configured to cooperate with the pump to circulate fluid.

15. The system of claim 1, further comprising a resilient body coupled to the fluid cooling
5 supply device, the resilient body having a fluid channel that extends through the body with at least a portion of the fluid channel arranged adjacent to the thermally conductive surface, wherein the fluid channel is in fluid communication with the energy delivery device.

10 16. The system of claim 15, further comprising a biasing device removably coupled to the resilient body, the biasing device configured to bias the resilient body with a given force to chill the fluid to a temperature below a selected temperature.

15 17. The system of claim 15, wherein the resilient bag includes a thermally conductive surface with at least a portion of the fluid channel arranged adjacent to the thermally conductive surface.

18. A method of treating a patient, comprising:
 providing a cooling system to chill and deliver fluid to the patient, the cooling
 system having a fluid heat exchanger;
 positioning an ablation assembly of a delivery device within an airway of the
20 patient such that the ablation assembly is apposed against a wall of the airway, the ablation assembly having an electrode;
 coupling the fluid heat exchanger to the ablation assembly to be in fluid communication with each other;
 chilling the fluid in the fluid heat exchanger with the cooling device; and
25 thermally treating a tissue by circulating the fluid from the fluid heat exchanger through the delivery device and simultaneously delivering energy to the electrode to treat the tissue positioned adjacent the airway of the patient.

19. The method of claim 18, wherein thermally treating tissue further comprises damaging
30 nerve tissue of a nerve trunk such that nervous system signals transmitted to a portion of the bronchial tree are attenuated.

20. The method of claim 18, further comprising drawing the fluid through the fluid heat exchanger at a negative pressure for supplying the fluid to the delivery device.

21. The method of claim 18, wherein treating tissue further comprises supplying the chilled fluid to the delivery device at a positive pressure.
- 5 22. The method of claim 18, wherein circulating the fluid includes supplying the fluid to the delivery device such that a temperature at the delivery device is maintained at or below 20°C during treatment of the patient.
- 10 23. The method of claim 18, wherein circulating the fluid includes supplying the fluid to the delivery device such that a temperature at the delivery device is maintained between 20°C and -5°C during treatment of the patient.
- 15 24. The method of claim 18, wherein circulating the fluid includes supplying the fluid to the delivery device such that a temperature at the delivery device is maintained between 5°C and -2°C during treatment of the patient.
- 20 25. The method of claim 18, wherein chilling the fluid includes transferring heat from the fluid in the heat exchanger to a temperature at or below 5°C during treatment of the patient.
- 25 26. The method of claim 18, wherein chilling the fluid includes transferring heat from the fluid in the fluid heat exchanger to a temperature between 5°C and -2°C during treatment of the patient.
27. The method of claim 18, further comprising maintaining a selected temperature at the delivery device for a selected interval during each of multiple tissue treatments of the patient.
28. The method of claim 27, wherein the selected interval for each tissue treatment is less than 60 seconds.
- 30 29. The method of claim 27, wherein the selected temperature is between 20°C and -5°C.
30. The method of claim 18, comprising pumping the fluid at a fluid flow rate of between 70 milliliters to 160 milliliters per minute.

31. The method of claim 18, comprising delivering the fluid to the patient at a pressure between 25 psi and 150 psi.
32. The method of claim 18, comprising biasing the fluid heat exchanger to a cooling device of the cooling system, the fluid heat exchanger comprising a resilient body having a fluid channel that extends through the body, wherein the fluid channel is in fluid communication with the ablation assembly.
33. The method of claim 18, comprising biasing the fluid heat exchanger to a cooling device of the cooling system, the fluid heat exchanger comprising a cartridge having a fluid channel that extends through the cartridge, wherein the fluid channel is in fluid communication with the ablation assembly.
34. A fluid cooling system for thermally treating fluid for treatment of a patient, comprising:
a cooling device that includes a thermal plate;
a heat exchanger removably coupled to the cooling device, the heat exchanger including a thermally conductive surface and a fluid channel that extends through the heat exchanger with at least a portion of the fluid channel arranged adjacent to the thermally conductive surface; and
at least one biasing mechanism arranged to removably couple the heat exchanger to the cooling device such that the thermally conductive surface is biased against the thermal plate of the cooling device to transfer heat from the fluid.
35. The system of claim 34, wherein the heat exchanger is a disposable heat exchanger cartridge, and the at least one biasing mechanism comprises two pairs of biasing mechanisms, each pair positioned on opposing ends of the cartridge.
36. The system of claim 35, wherein the cartridge is formed to be in a first state when disengaged from the cooling device and in a second state when engaged to the cooling device, wherein the first state includes the thermally conductive surface of the cartridge having a first profile with a convex shape relative to the thermal plate, and wherein the second state includes the thermally conductive surface of the cartridge having a second profile that is a substantially flat shape relative to the thermal plate such that the thermally conductive surface of the cartridge

and the thermal plate are substantially biased against each other to effectuate heat transfer from the fluid.

37. The system of claim 35, wherein the cooling device includes a perimeter portion that includes at least one corresponding biasing mechanism that is coupled to the at least one biasing mechanism of the cartridge when the cartridge is engaged to the cooling device.

38. The system of claim 37, wherein the at least one biasing mechanism and the at least one corresponding biasing mechanism are each comprised of a plurality of magnets attractable to each other such that a given biasing force is applied between the cartridge and the cooling device to effectuate heat transfer from the fluid.

39. The system of claim 35, wherein the cartridge includes a first plate and a second plate coupled to each other, wherein the first plate includes the thermally conductive surface having a thickness at least .01 inch or less.

40. The system of claim 39, wherein the second plate includes a thermally insulating material and includes a groove defining the fluid channel.

41. The system of claim 35, wherein the fluid channel includes at least one corner portion proximate a transition between a first sidewall and a second side wall of the fluid channel, the at least one corner portion configured such that gas bubbles are not trapped proximate the at least one corner portion during operation of the system.

42. The system of claim 35, wherein the cartridge further comprises an input port and an output port in fluid communication with each other and with the fluid channel for supplying fluid to a treatment device positioned in the patient.

43. The system of claim 35, further comprising a variable volume reservoir contained in the cartridge such that the fluid is drawn from an outlet of the reservoir and supplied to the patient.

44. The system of claim 35, further comprising a self-aligning means for automatic alignment of the cartridge and automatic biasing of the cartridge to the cooling device.

45. The system of claim 34, further comprising at least one controller coupled to the cooling device for regulating an amount of heat transfer from the fluid.
46. The system of claim 34, further comprising a pump configured to regulate a volume of fluid circulating through the system, wherein the pump is positioned adjacent the heat exchanger and the heat exchanger includes a negative fluid pressure.
47. The method of claim 46, wherein the pump is configured to circulate the fluid at a fluid flow rate of between 70 milliliters to 160 milliliters per minute.
48. The method of claim 46, wherein the pump is configured to deliver the fluid to the patient at a pressure between 25 psi and 150 psi.
49. The method of claim 46, further comprising a pump positioned in a return path downstream of a treatment device positioned in the patient, the pump configured to draw the fluid through the treatment device from the heat exchanger.
50. The system of claim 34, further comprising a supply line in fluid communication with the fluid channel of the heat exchanger and a treatment device positioned in the patient.
51. The system of claim 34, further comprising a supply line in fluid communication with the fluid channel, wherein at least a portion of the supply line is positioned inside the patient and in fluid communication with a treatment device positioned inside a bronchus of the patient.
52. The system of claim 34, further comprising a pulmonary treatment system having a bronchoscope and a treatment device, the treatment device positioned adjacent to pulmonary tissue of the patient, the heat exchanger in fluid communication with the treatment device to supply chilled fluid to the patient during treatment.
53. The system of claim 34, wherein the heat exchanger is a resilient body.
54. The system of claim 53, wherein the at least one biasing mechanism is a plate removably coupled to the resilient body, the plate removably coupled to the resilient body such that the resilient body is positioned between the plate and the thermal plate.

55. The system of claim 53, wherein the resilient body comprises an input port and an output port in fluid communication with each other and with the fluid channel for supplying fluid to a treatment device positioned in the patient.

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56. A disposable heat exchanger cartridge for thermally treating fluid for treatment of a patient, the cartridge comprising:

a thermally conductive surface and a fluid channel that extends through the cartridge with at least a portion of the fluid channel arranged adjacent to the thermally conductive surface; and

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at least one biasing mechanism arranged to removably couple the cartridge to a cooling device such that the thermally conductive surface conductively cools the fluid.

57. The cartridge of claim 56, wherein the at least one biasing mechanism comprises two pairs of magnets, each pair positioned on opposing ends of the cartridge and each pair attractable to the cooling device to improve surface-to-surface contact between the cartridge and the cooling device to increase thermal exchange efficiency.

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58. The cartridge of claim 56, wherein the cartridge is formed to be in a first state when disengaged from the cooling device and in a second state when engaged to the cooling device.

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59. The cartridge of claim 58, wherein the first state includes the cartridge having a substantially convex shaped profile, and wherein the second state includes the cartridge having a substantially rectangular shaped profile.

25

60. The cartridge of claim 56, wherein the at least one biasing mechanism is comprised of a plurality of magnets attractable to the cooling device such that a given biasing force is applied to the cartridge to effectuate heat transfer from the fluid.

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61. The cartridge of claim 56, further comprising a first plate and a second plate coupled to each other, wherein the first plate includes the thermally conductive surface, the first plate having a thickness at least .01 inch or less.

62. The cartridge of claim 61, wherein the second plate is comprised of insulation material and includes a groove defining the fluid channel.
63. The cartridge of claim 56, wherein the fluid channel includes at least one corner portion proximate a transition between a first sidewall and a second side wall of the fluid channel, the at least one corner portion configured such that gas bubbles are not trapped proximate the at least one corner portion during operation of the system.
64. The cartridge of claim 56, further comprising an input port and an output port in fluid communication with each other and with the fluid channel.
65. The cartridge of claim 56, further comprising a variable volume reservoir contained in the cartridge such that the fluid is drawn from the reservoir and supplied to the patient.
66. The cartridge of claim 56, further comprising a self-aligning means for automatic alignment and biasing of the cartridge when engaged to the cooling device.
67. A fluid cooling system for thermally treating fluid for treatment of a patient, comprising:
a cooling device that includes a thermal plate;
a disposable heat exchanger cartridge removably coupled to the cooling device, the cartridge including a thermally conductive surface and a fluid channel that extends through the cartridge with at least a portion of the fluid channel arranged adjacent to the thermally conductive surface; and
at least one biasing mechanism arranged to removably couple the cartridge to the cooling device such that the thermally conductive surface is biased against the thermal plate of the cooling device to transfer heat from the fluid.
68. The system of claim 67, wherein the at least one biasing mechanism includes a cam system operable between a first position for engaging the cartridge to the cooling device and a second position for disengaging the cartridge from the cooling device.
69. The system of claim 68, wherein the cam system includes a cam lever and a cam shaft having at least one cam lobe.

70. The system of claim 68, wherein the first position of the cam system is a locked configuration such that the thermally conductive surface of the cartridge is biased to the thermal plate to effectuate heat transfer, and wherein the second position is an unlocked configuration such that the cartridge can be removed from the cooling device.

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71. The system of claim 67, further comprising a front plate coupled to the cooling device and the cam system, the front plate having a slot sized to slideably receive the cartridge.

72. The system of claim 71, wherein the front plate includes an upper biasing surface and a lower biasing surface, the upper and lower biasing surfaces each being non-parallel to the thermal plate, and wherein the cartridge includes a corresponding upper angled surface and a corresponding lower angled surface, the corresponding upper and lower angled surfaces each being non-parallel to the thermal plate and each being respectively parallel to the upper and lower biasing surfaces of the front plate, such that the cartridge is slideably received in the slot.

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73. The system of claim 72, wherein the at least one biasing mechanism includes a cam system operable between a first position for engaging the cartridge to the cooling device and a second position for disengaging the cartridge from the cooling device, wherein the first position includes the cam system biasing the upper biasing surface of the front plate such that the thermally conductive surface and the thermal plate are biased to each other at a given force.

20

74. The system of claim 67, wherein the cartridge includes a first plate and a second plate coupled to each other, wherein the first plate includes the thermally conductive surface and the second plate includes a thermally insulating material and a groove defining the fluid channel.

25

75. The system of claim 67, wherein the fluid channel includes at least one corner portion proximate a transition between a first sidewall and a second side wall of the fluid channel, the at least one corner portion configured such that gas bubbles are not trapped proximate the at least one corner portion during operation of the system.

30

76. The system of claim 74, wherein the second plate includes a variable volume reservoir such that the fluid is drawn from the reservoir supplied to the patient.

77. The system of claim 67, further comprising a pump configured to regulate a volume of fluid circulating through the system, wherein the pump is positioned adjacent the cartridge such that the fluid in the cartridge has a negative fluid pressure.

5 78. The system of claim 77, wherein the pump is configured to circulate the fluid at a fluid flow rate of between 70 milliliters to 160 milliliters per minute.

79. The system of claim 77, wherein the pump is configured to deliver the fluid at a pressure between 25 psi and 150 psi.

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80. The system of claim 67, further comprising a pump positioned downstream of a treatment device positioned in the patient, the pump configured to draw the fluid through the treatment device.

15 81. The system of claim 67, further comprising a supply line in fluid communication with the fluid channel of the cartridge and a treatment device positioned in the patient.

82. The system of claim 67, further comprising a supply line in fluid communication with the fluid channel, wherein at least a portion of the supply line is positioned inside the patient and in
20 fluid communication with a treatment device positioned inside a bronchus of the patient.

83. The system of claim 67, further comprising a pulmonary treatment system having a bronchoscope and a treatment device, the treatment device positioned adjacent to pulmonary tissue of the patient, the cartridge in fluid communication with the treatment device to supply
25 chilled fluid to the patient during treatment.

84. A method of cooling a fluid for treatment of a patient, comprising:

 biasing a heat exchanger cartridge to a thermal plate of a cooling device, the heat exchanger cartridge having a fluid channel and at least one biasing mechanism to
30 removably attach the heat exchanger cartridge to the cooling device;

 removing the heat exchanger cartridge from the cooling device; and

 biasing a replacement heat exchanger cartridge to the thermal plate of the cooling device, the replacement heat exchanger cartridge having a fluid channel and at least one

biasing mechanism to removably attach the replacement heat exchanger cartridge to the cooling device.

5 85. The method of claim 84, wherein biasing the cartridges includes applying a given biasing force between the cartridges and the cooling device by utilizing magnetic force.

86. The method of claim 85, wherein the given biasing force is at least 10 pounds of force.

10 87. The method of claim 85, wherein the given biasing force is between 10 and 60 pounds of force.

15 88. The method of claim 84, wherein biasing the cartridges includes biasing a thermally conductive surface of the cartridges to the thermal plate, wherein at least a portion of the thermally conductive surface is adjacent to the fluid channel of the cartridges.

89. The method of claim 84, further comprising pumping the fluid through the heat exchanger cartridge for fluid delivery to the patient.

20 90. The method of claim 84, further comprising supplying chilled fluid to a treatment device positioned adjacent to pulmonary tissue of the patient during a pulmonary treatment.

91. The method of claim 84, wherein biasing the heat exchanger cartridge further comprises actuating a cam system to an engaged state to bias the cartridge to the thermal plate.

25 92. The method of claim 84, wherein removing the heat exchanger cartridge comprises actuating the cam system to a disengaged state to disengage the cartridge from the thermal plate.

30 93. The method of claim 91, wherein actuating the cam system comprises moving a cam lever and a cam shaft to the engaged state to bias the cartridge to the thermal plate.

94. A fluid cooling system for thermally treating fluid for treatment of a patient, comprising:
a cooling device that includes a thermal plate;
a heat exchanger coupled to the cooling device, the heat exchanger including a thermally conductive surface and a fluid channel that extends through the heat exchanger

with at least a portion of the fluid channel arranged adjacent to the thermally conductive surface; and

a pump positioned at a downstream side of the fluid channel of the heat exchanger, the pump adapted to draw fluid through the fluid channel at a negative pressure to provide the fluid to a treatment device at a positive pressure.

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95. The system of claim 94, wherein the pump includes a forward gear and a reverse gear, the reverse gear adapted to reverse a flow of the fluid through the heat exchanger to remove gas from the system.

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96. The system of claim 94, wherein the pump provides a fluid flow rate of between 70 milliliters to 160 milliliters per minute.

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97. The system of claim 94, wherein the pump is configured to deliver the fluid at a pressure between 25 psi and 150 psi.

98. The system of claim 94, wherein the heat exchanger is positioned substantially vertical relative to the thermal plate.

20

99. The system of claim 94, wherein the positive pressure is at least 80 psi.

100. The system of claim 94, wherein the fluid returned to the heat exchanger is 10 psi or less.

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101. The system of claim 94, further comprising a supplemental pump positioned downstream the treatment device, the pump configured to draw the fluid through the treatment device from the heat exchanger.

30

102. The system of claim 94, wherein the heat exchanger is a cartridge having a first plate and a second plate coupled to each other, wherein the first plate includes the thermally conductive surface and the second plate includes a groove defining the fluid channel.

103. The system of claim 102, wherein the fluid channel includes at least one corner portion proximate a transition between a first sidewall and a second side wall of the fluid channel, the at

least one corner portion configured such that gas bubbles are not trapped proximate the at least one corner portion during operation of the system.

5 104. The system of claim 94, wherein the heat exchanger is a resilient body having a membrane, wherein the membrane is the thermally conductive surface.

105. The system of claim 104, wherein the resilient body includes a serpentine groove defining the fluid channel.

10 106. A method of cooling fluid for treatment of a patient, comprising:
drawing a coolant through a heat exchanger at a negative pressure to chill the coolant;
supplying the chilled coolant to a treatment device positioned inside the patient at a positive pressure; and
15 circulating the coolant through the treatment device.

107. The method of claim 106, wherein the positive pressure is at least 80 psi.

20 108. The method of claim 106, wherein a return pressure to the heat exchanger is 10 psi or less.

109. The method of claim 106, further comprising regulating an amount of heat transfer from the fluid.

25 110. The method of claim 106, further comprising regulating an amount of volume of the fluid to supply for treatment of the patient.

30 111. The method of claim 106, further comprises biasing the heat exchanger to a cooling device.

112. The method of claim 106, further comprising positioning the treatment device inside a bronchus of the patient to receive the circulated chilled fluid.

113. The method of claim 106, further comprising

positioning a pump at a downstream side of the heat exchanger; and
pumping the fluid with the pump through the heat exchanger in a reverse manner
to substantially remove gas from the heat exchanger.

- 5 114. The method of claim 106, wherein supplying the chilled coolant to the treatment device includes pushing the chilled coolant with a pump, and wherein circulating the chilled coolant includes drawing the coolant from the treatment device with a supplemental pump.
115. The method of claim 106, wherein the chilled coolant is circulated at a fluid flow rate of
10 between 70 milliliters to 160 milliliters per minute.
116. The method of claim 106, wherein the chilled fluid is supplied to the treatment device at a pressure between 25 psi and 150 psi.
- 15 117. The method of claim 106, comprising biasing the heat exchanger to a cooling device, wherein the heat exchanger is a cartridge having a fluid channel in fluid communication with the treatment device.
118. The method of claim 106, comprising biasing the heat exchanger to a cooling device,
20 wherein the heat exchanger is a resilient body having a fluid channel in fluid communication with the treatment device.
119. A fluid cooling system for thermally treating fluid for treatment of a patient, comprising:
a cooling device that includes a thermal plate;
25 a resilient body removably coupled to the cooling device, the body including a thermally conductive surface and a fluid channel that extends through the body with at least a portion of the fluid channel arranged adjacent to the thermally conductive surface;
and
at least one biasing mechanism arranged to removably couple the body to the
30 cooling device such that the thermally conductive surface is biased against the thermal plate of the cooling device to transfer heat from the fluid.

120. The system of claim 119, wherein the at least one biasing mechanism includes a cam system operable between a first position for engaging the body to the cooling device and a second position for disengaging the body from the cooling device.

5 121. The system of claim 120, wherein the cam system includes a cam lever and a cam shaft having at least one cam lobe.

122. The system of claim 120, wherein the first position of the cam system is a locked configuration such that the thermally conductive surface of the body is biased to the thermal
10 plate to effectuate heat transfer, and wherein the second position is an unlocked configuration such that the body can be removed from the cooling device.

123. The system of claim 119, wherein the at least one biasing mechanism includes a cam system operable between a first position for engaging the body to the cooling device and a
15 second position for disengaging the body from the cooling device, wherein the first position includes the cam system biasing the body to the thermal plate with a given force.

124. The system of claim 119, further comprising a pump configured to regulate a volume of fluid circulating through the system, wherein the pump is positioned adjacent the body such that
20 the fluid in the body has a negative fluid pressure.

125. The system of claim 124, wherein the pump is configured to circulate the fluid at a fluid flow rate of between 70 milliliters to 160 milliliters per minute.

25 126. The system of claim 124, wherein the pump is configured to deliver the fluid at a pressure between 25 psi and 150 psi.

127. The system of claim 119, further comprising a supply line in fluid communication with the fluid channel of the body and a treatment device positioned in the patient.

30 128. The system of claim 119, further comprising a supply line in fluid communication with the fluid channel, wherein at least a portion of the supply line is positioned inside the patient and in fluid communication with a treatment device positioned inside a bronchus of the patient.

129. The system of claim 119, further comprising a pulmonary treatment system having a bronchoscope and a treatment device, the treatment device positioned adjacent to pulmonary tissue of the patient, the body in fluid communication with the treatment device to supply chilled fluid to the patient during treatment.

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130. A method of cooling a fluid for treatment of a patient, comprising:

 biasing a resilient body to a thermal plate of a cooling device, the body having a fluid channel, wherein to body is biased to the thermal plate by at least one biasing mechanism;

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 removing the body from the cooling device; and

 biasing a replacement resilient body to the thermal plate of the cooling device with the at least one biasing mechanism.

131. The method of claim 130, wherein biasing the body includes applying a given biasing force to the body.

15

132. The method of claim 131, wherein the given biasing force is at least 10 pounds of force.

133. The method of claim 131, wherein the given biasing force is between 10 and 60 pounds of force.

20

134. The method of claim 130, further comprising pumping the fluid through the body for fluid delivery to the patient.

135. The method of claim 130, further comprising supplying chilled fluid to a treatment device positioned adjacent to pulmonary tissue of the patient during a pulmonary treatment.

25

136. The method of claim 130, wherein biasing the body further comprises actuating a cam system to an engaged state to bias the body to the thermal plate.

30

137. The method of claim 130, wherein removing the body comprises actuating the cam system to a disengaged state to disengage the body from the thermal plate.

138. The method of claim 137, wherein actuating the cam system comprises moving a cam lever and a cam shaft to the engaged state to bias the body to the thermal plate.

5 139. A disposable heat exchanger cartridge for thermally treating fluid for treatment of a patient, the cartridge comprising:

a plate having a thermally conductive surface; and

a thermoformed tray having a recess or groove defined therein, wherein the thermoformed tray is coupled to the thermally conductive surface, thereby defining a fluid channel that extends through the cartridge, and

10 wherein the cartridge is configured to be removably couplable to a cooling device a cooling device such that the thermally conductive surface conductively cools the fluid.

140. The cartridge of claim 139, wherein the plate has a thickness at least .01 inch or less.

15 141. The cartridge of claim 140, wherein the plate comprises copper, and wherein the thermally conductive surface comprises a silver layer.

142. The cartridge of claim 141, wherein the silver layer has a thickness of about 0.5 to 1 micron.

20 143. The cartridge of claim 139, wherein the cartridge further comprises an inlet port coupled to a first end of the fluid channel, and in fluid communication with the fluid channel and a fluid reservoir external to the cartridge, and an outlet port coupled to a second end of the fluid channel, and in fluid communication with the fluid channel and a pumping mechanism external
25 to the cartridge.

144. The cartridge of claim 143, wherein the inlet port and the outlet port each have a sloped or trapezoidal cross-sectional profile such that gas bubbles are trapped in at least one of the ports during operation of the system.

30 145. The cartridge of claim 139, wherein at least one edge of the cartridge has self-aligning means for automatic alignment of the cartridge when engaged to a cooling device.

146. The cartridge of claim 139, wherein the fluid channel serpentine along a length and a width of the cartridge forming a plurality of passes.

147. The cartridge of claim 139, wherein the tray is formed of a thermoformable material comprising polyvinyl chloride (PVC) or polyethylene terephthalate (PET).

148. A fluid cooling system for thermally treating fluid for treatment of a patient, comprising:
a housing including a hinged door shiftable between an open position and a closed position;

10 a cooling device mounted at least partially within the housing, the cooling device including a thermal plate;

a disposable heat exchanger cartridge removably coupled to the cooling device, the cartridge including a plate having a thermally conductive surface, and a thermoformed tray having a recess or groove defined therein, wherein the thermoformed tray is coupled to the thermally conductive surface, thereby defining a fluid channel that extends through the cartridge; and

15 at least one biasing mechanism arranged to removably couple the cartridge to the cooling device such that the thermally conductive surface is biased against the thermal plate of the cooling device to transfer heat from the fluid.

20 149. The system of claim 148, wherein the at least one biasing mechanism comprises one or more magnets that are attractable to the cooling plate such that that a given biasing force is applied between the cartridge and the cooling device to effectuate heat transfer from the fluid.

25 150. The system of claim 148, further comprising a pump configured to regulate a volume of fluid circulating through the system, wherein the pump is positioned adjacent the cartridge such that the fluid in the cartridge has a negative fluid pressure.

151. The system of claim 148, wherein the pump is configured to circulate the fluid at a fluid flow rate of between 70 milliliters to 160 milliliters per minute.

30 152. The system of claim 151, wherein the pump is configured to deliver the fluid at a pressure between 25 psi and 150 psi.

153. The system of claim 148, further comprising a pump positioned downstream of a treatment device positioned in the patient, the pump configured to draw the fluid through the treatment device.

5 154. The system of claim 148, further comprising a supply line in fluid communication with the fluid channel of the cartridge and a treatment device positioned in the patient.

155. The system of claim 148, further comprising a supply line in fluid communication with the fluid channel, wherein at least a portion of the supply line is positioned inside the patient and
10 in fluid communication with a treatment device positioned inside a bronchus of the patient.

156 The system of claim 148, further comprising a pulmonary treatment system having a bronchoscope and a treatment device, the treatment device positioned adjacent to pulmonary tissue of the patient, the cartridge in fluid communication with the treatment device to supply
15 chilled fluid to the patient during treatment.

157. The system of claim 148, further comprising a variable volume fluid reservoir in fluid communication with the fluid channel via a reservoir supply line.

20 158. The system of claim 148, wherein the system is a closed-loop system in which fluid is pumped from a fluid reservoir and back to the reservoir after circulation through a device in a patient.

159. The system of claim 158, wherein the fluid reservoir is in fluid communication with a
25 fluid supply line and a fluid return line in a single location on the reservoir via a coaxial bag spike assembly.

160. The system of claim 159, wherein the coaxial bag spike assembly is comprises a first tubular member having a first lumen therethrough and a first internal diameter, and a second
30 tubular member having a second lumen therethrough and a second internal diameter smaller than the first diameter, wherein the second tubular member is positioned within the first lumen of the first tubular member, such that the first lumen and the second lumen are coaxial and thereby defining a first channel and a second channel.

161. The system of claim 160, wherein one of the first and second channels is in fluid communication with a fluid return line such that used coolant is returned to the reservoir via the channel, and wherein the other of the first and second channels is in fluid communication with a fluid supply line such that coolant is supply from the reservoir to the fluid channel.

5

162. The system of claim 161, wherein each of the first and second lumens are coupled to the respective supply line and return line via a spike adapter, wherein the adapter comprises a tubular member coupled to a first end of the first tubular member, the adapter having a first port for coupling to the fluid return line such that the fluid return line and one of the first and second channels are in fluid communication, and a second port for coupling to the fluid supply line such that the fluid supply line and the other of the first and second channels are in fluid communication.

10

15

163. The system of claim 160, wherein the second tubular member is a hypotube, and the first tubular member is a non-vented female luer.

164. A coaxial bag spike assembly comprising:

a first tubular member having structure defining a first lumen therethrough, the first lumen having a first internal diameter; and

20

a second tubular member having structure defining a second lumen therethrough, the second lumen having a second internal diameter smaller than the first diameter,

wherein the second tubular member is positioned within the first lumen of the first tubular member, such that the first lumen and the second lumen are coaxial and thereby defining a first channel and a second channel, and

25

wherein one of the first and second channels is in fluid communication with a fluid return line of a cooling device and a fluid reservoir such that coolant is returned to the reservoir via the lumen, and wherein the other of the first and second channels is in fluid communication with a fluid supply line and the fluid reservoir such that coolant is supplied from the reservoir to the cooling device.

30

165. The assembly of claim 164, wherein the assembly further comprises a spike adapter for coupling the one of the supply line and return line to the first channel, and the other of the supply line and return line to the second channel, the spike adapter including a tubular member coupled to a first end of the first tubular member, the adapter having a first port for coupling to the fluid

return line such that the fluid return line and one of the first and second channels are in fluid communication, and a second port for coupling to the fluid supply line such that the fluid supply line and the other of the first and second channels are in fluid communication.

5 166. The assembly of claim 164, wherein the second tubular member is a hypotube, and the first tubular member is a non-vented female luer.

167. The assembly of claim 164, wherein the first tubular member is injection molded plastic.

10 168. The assembly of claim 165, wherein the spike adapter is injection molded plastic.

169. The assembly of claim 164, further comprising a vented spike cap for coupling the first and second tubular members to the fluid reservoir.

15

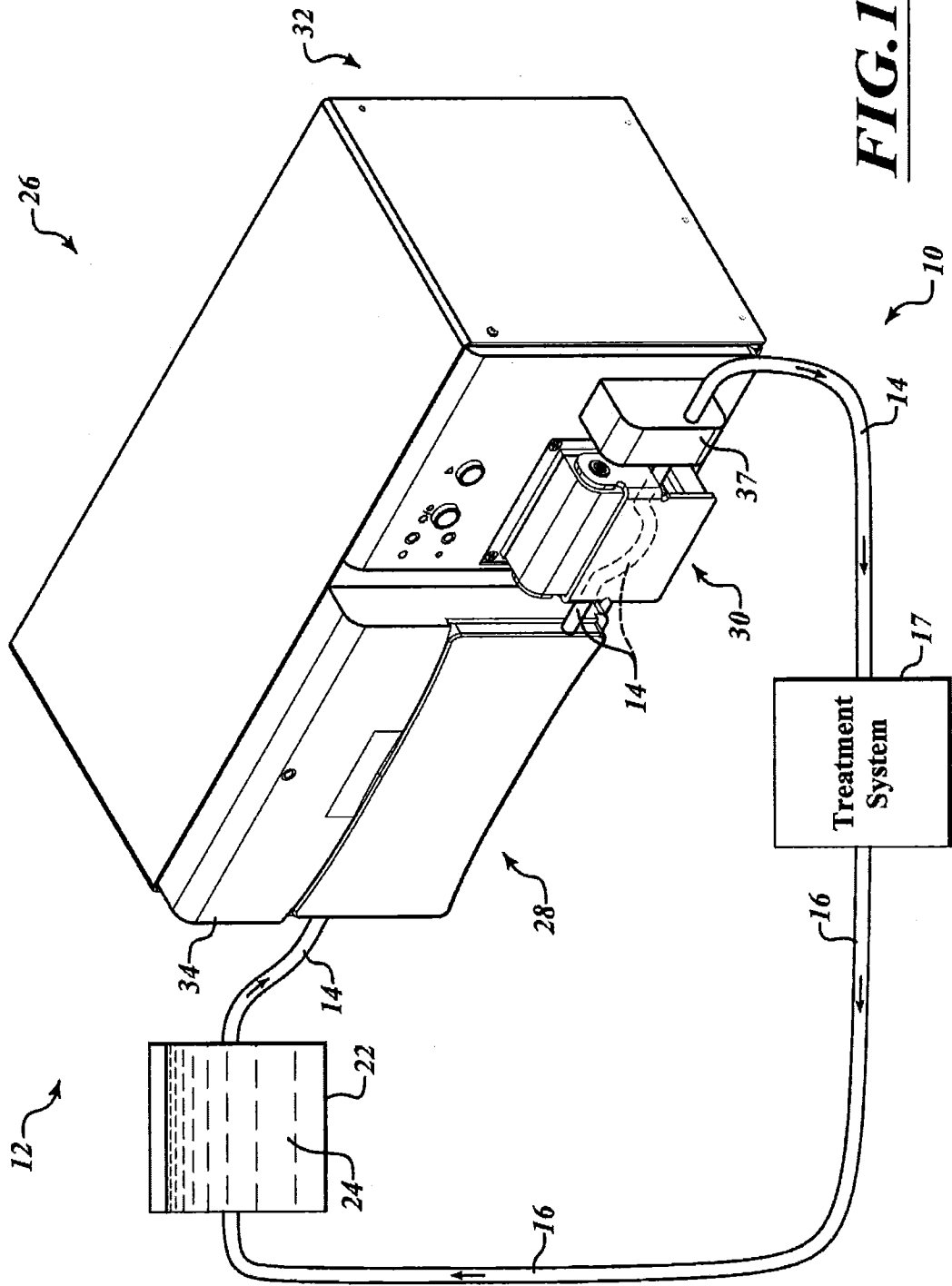


FIG. 1

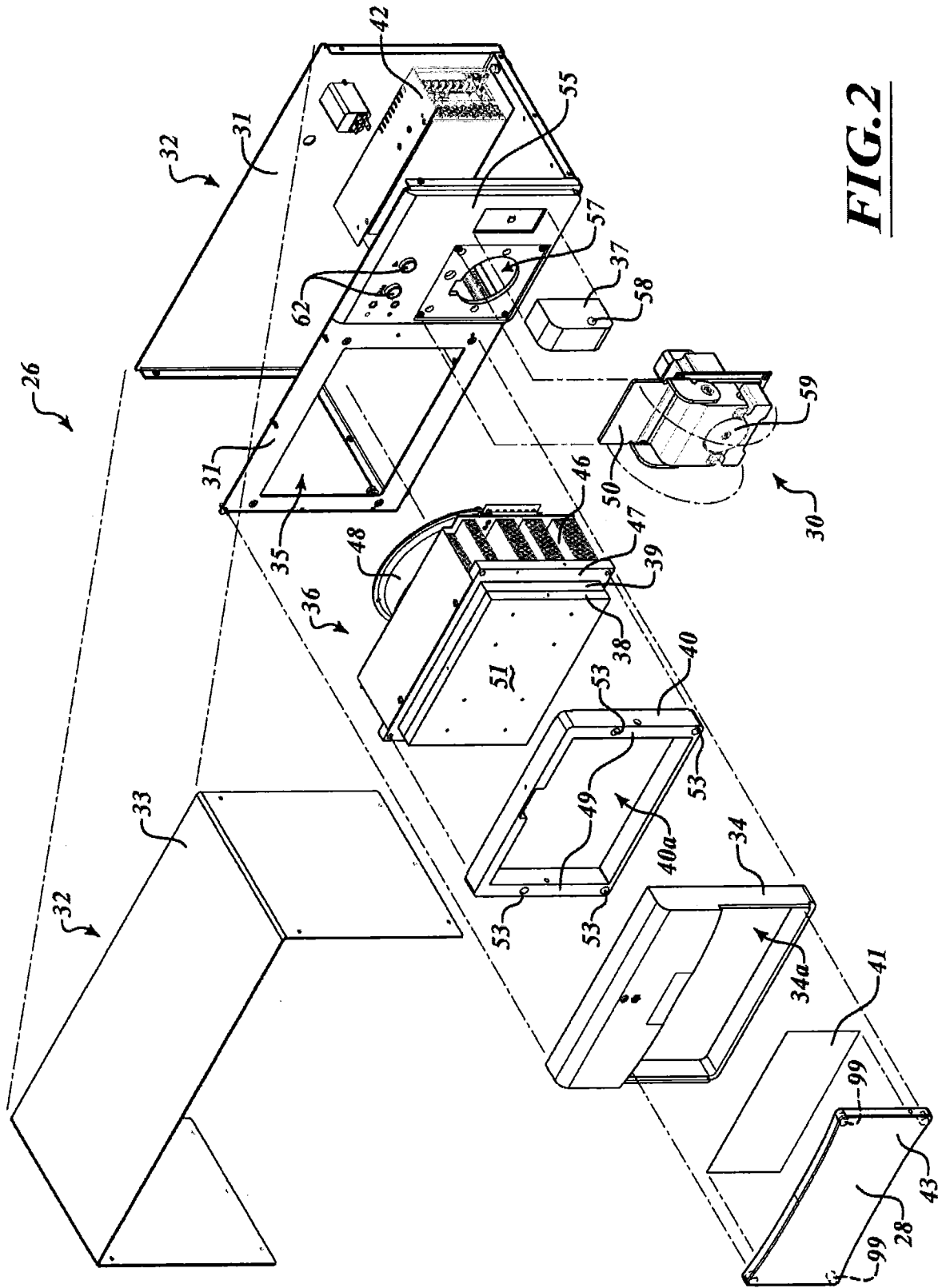


FIG. 2

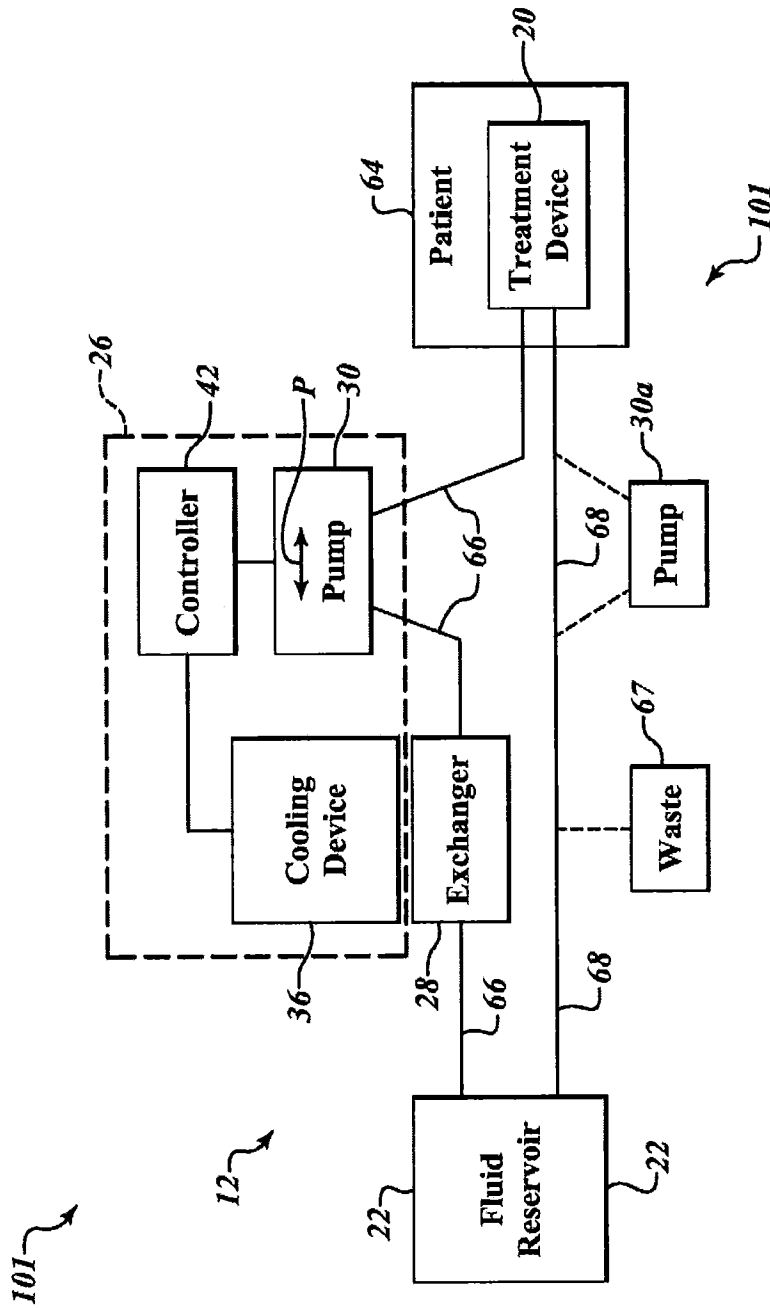
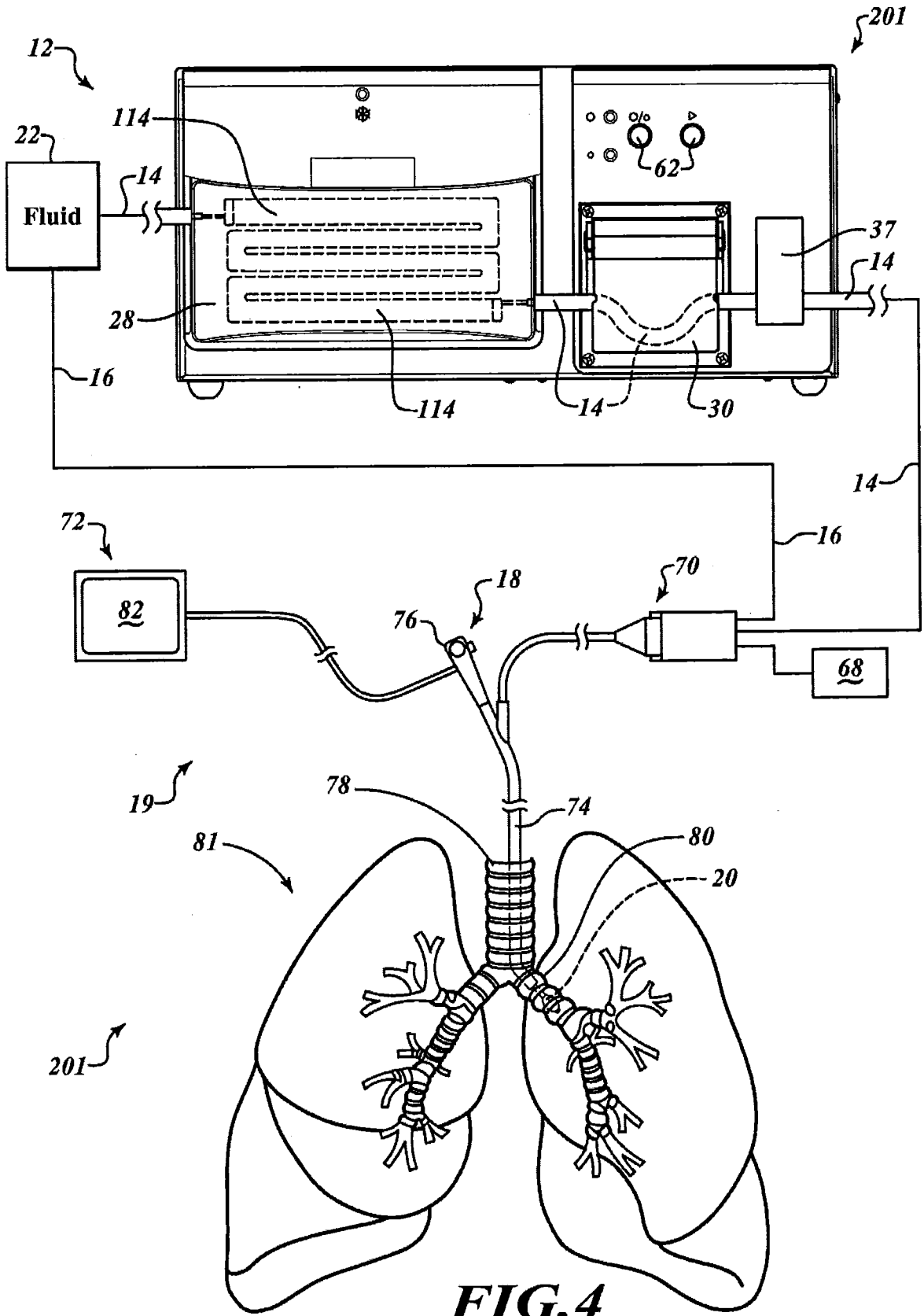


FIG.3



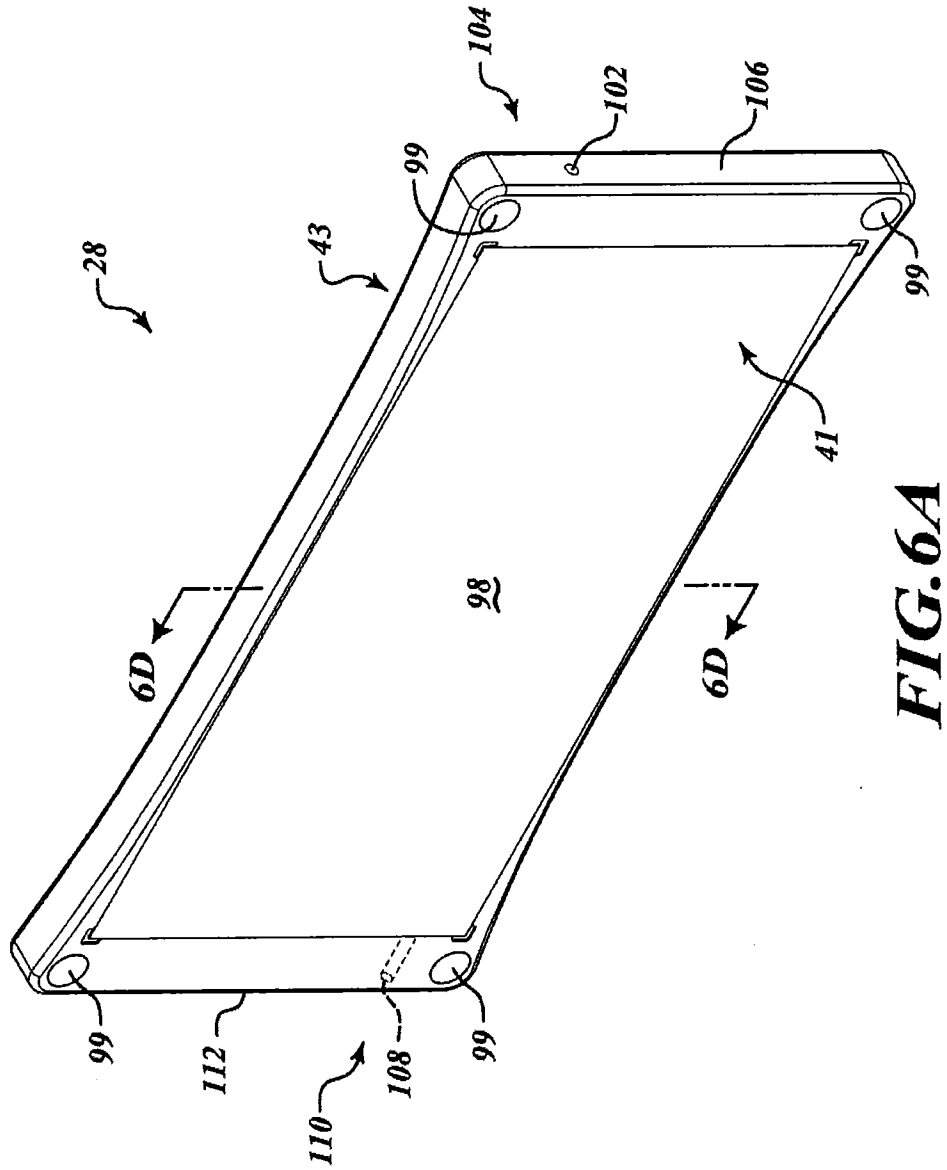


FIG. 6A

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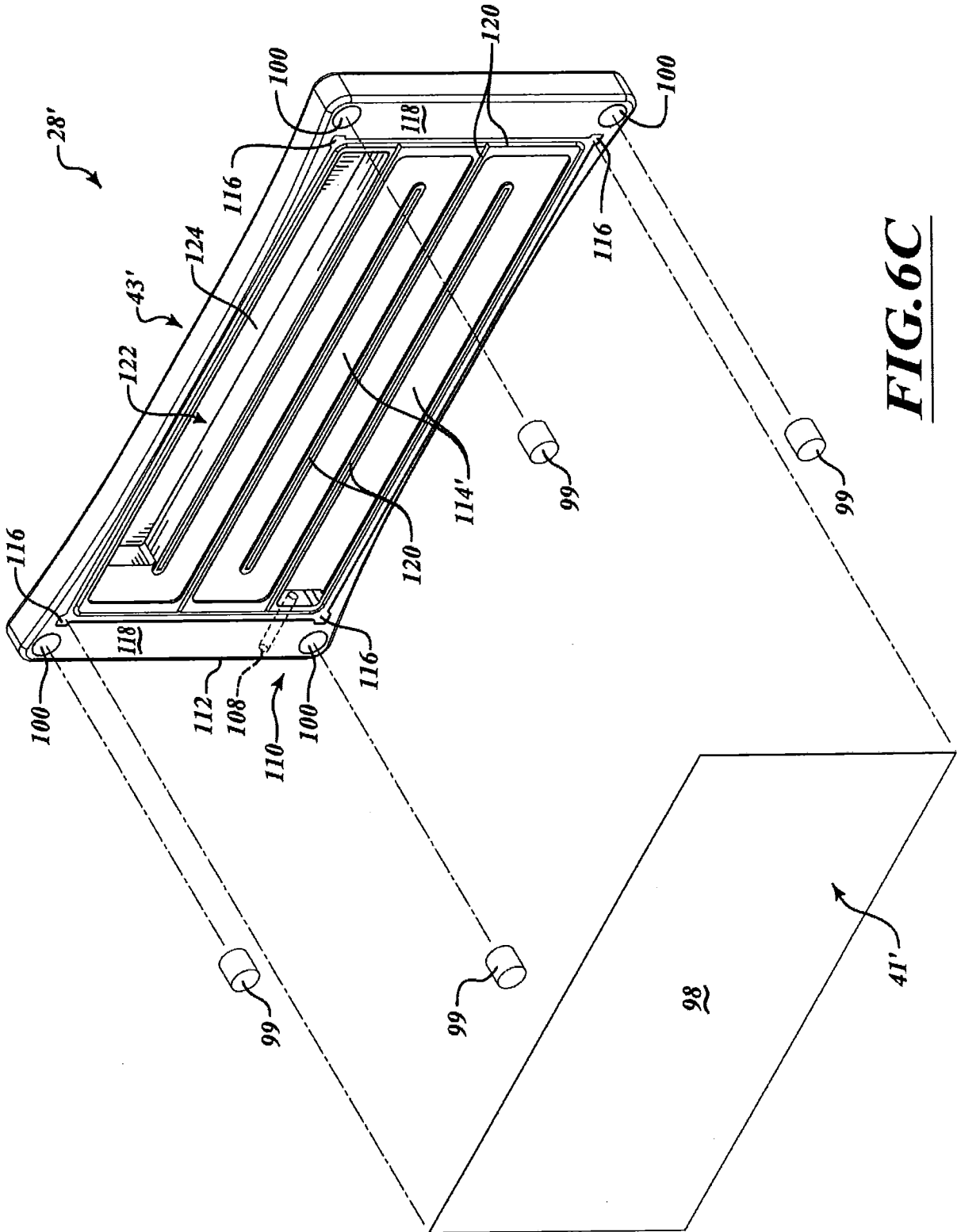


FIG. 6C

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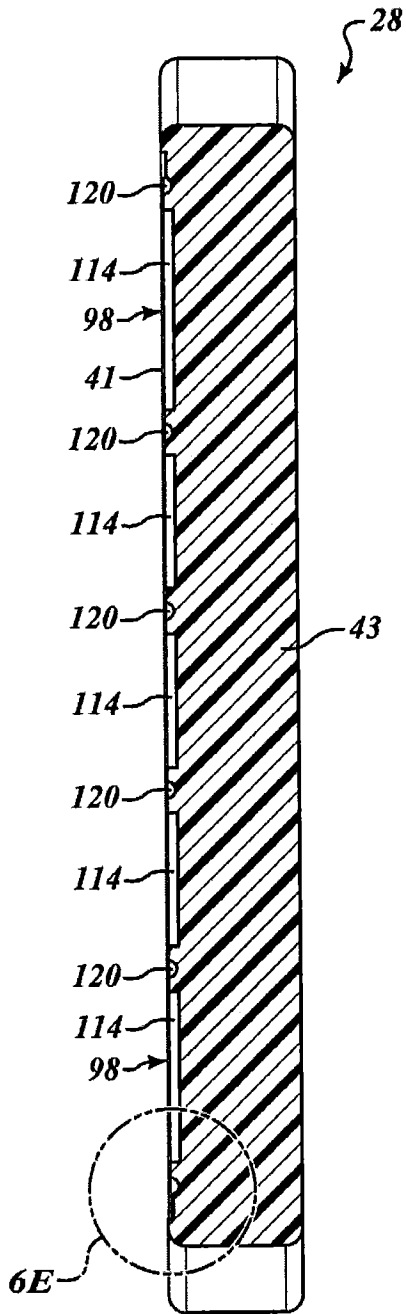


FIG. 6D

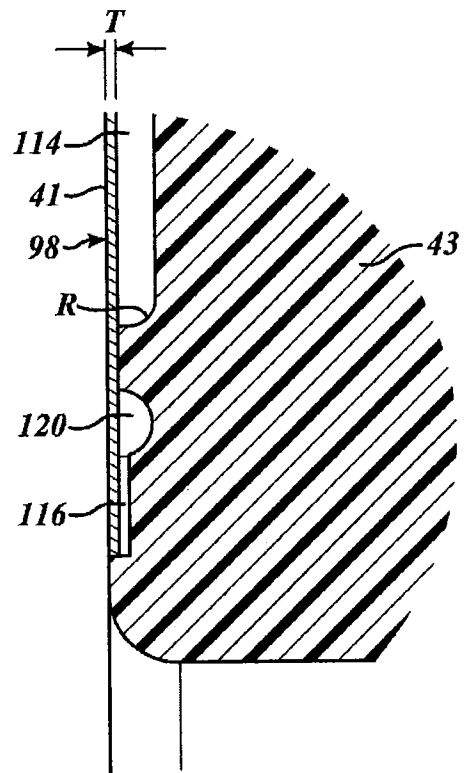


FIG. 6E

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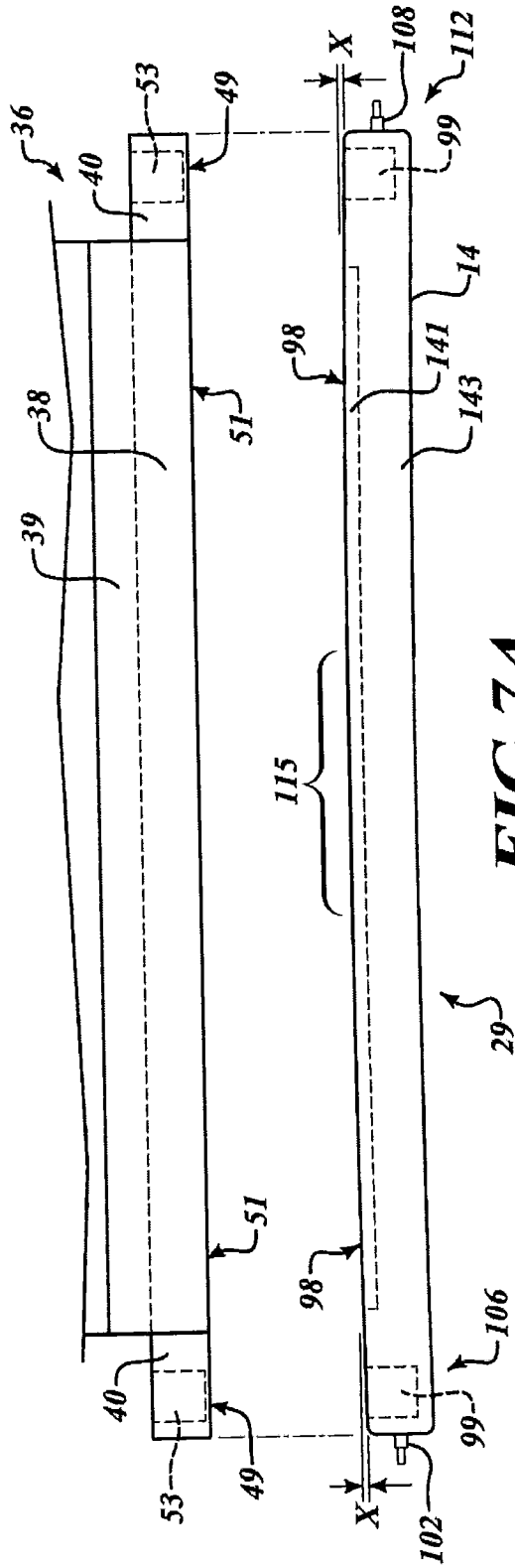


FIG. 7A

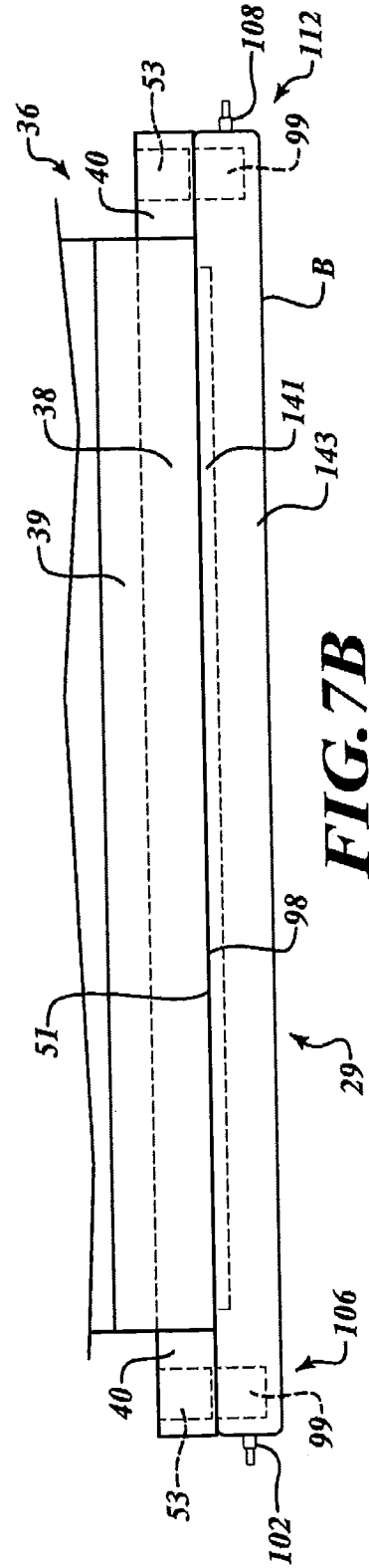
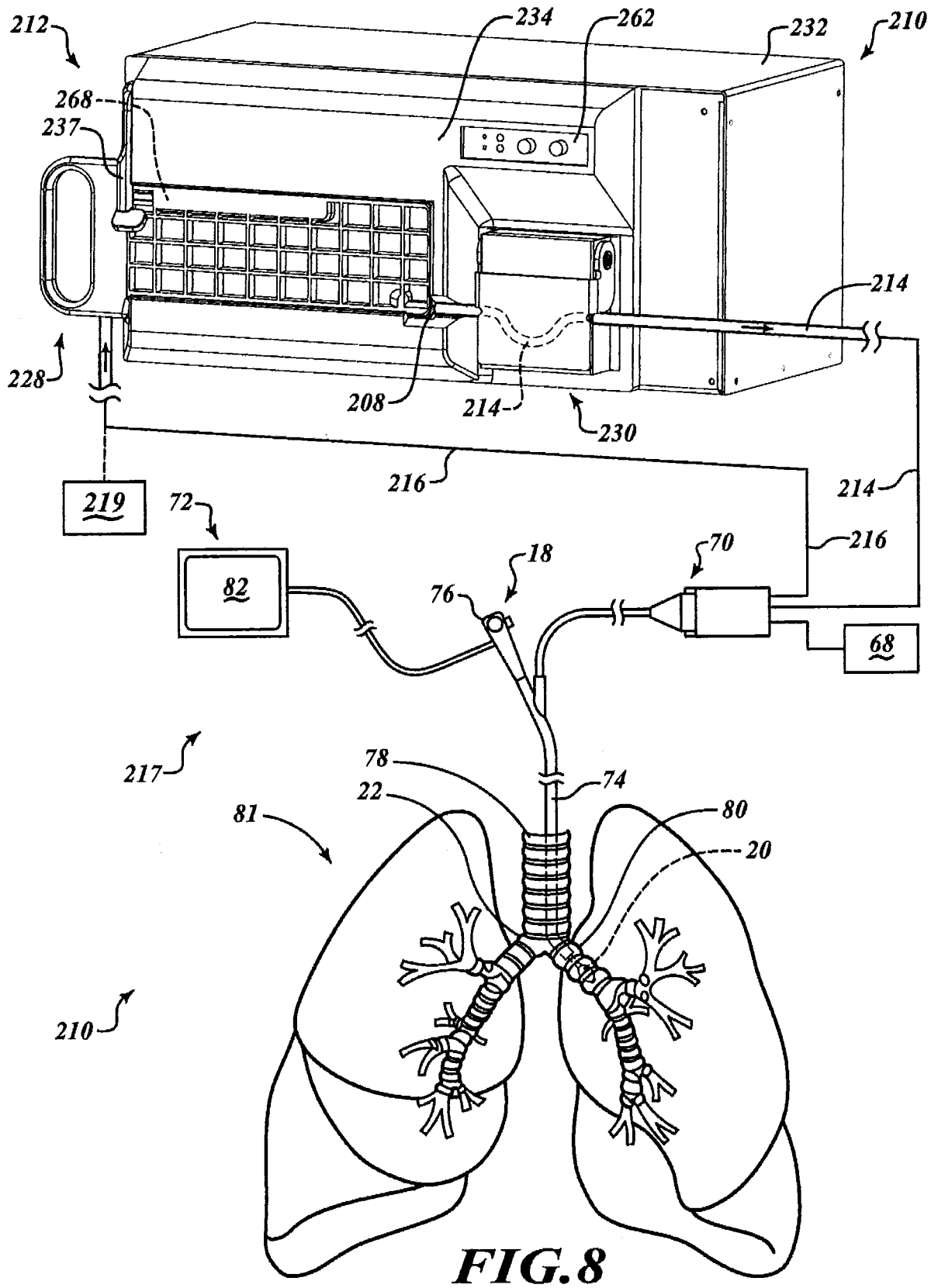


FIG. 7B

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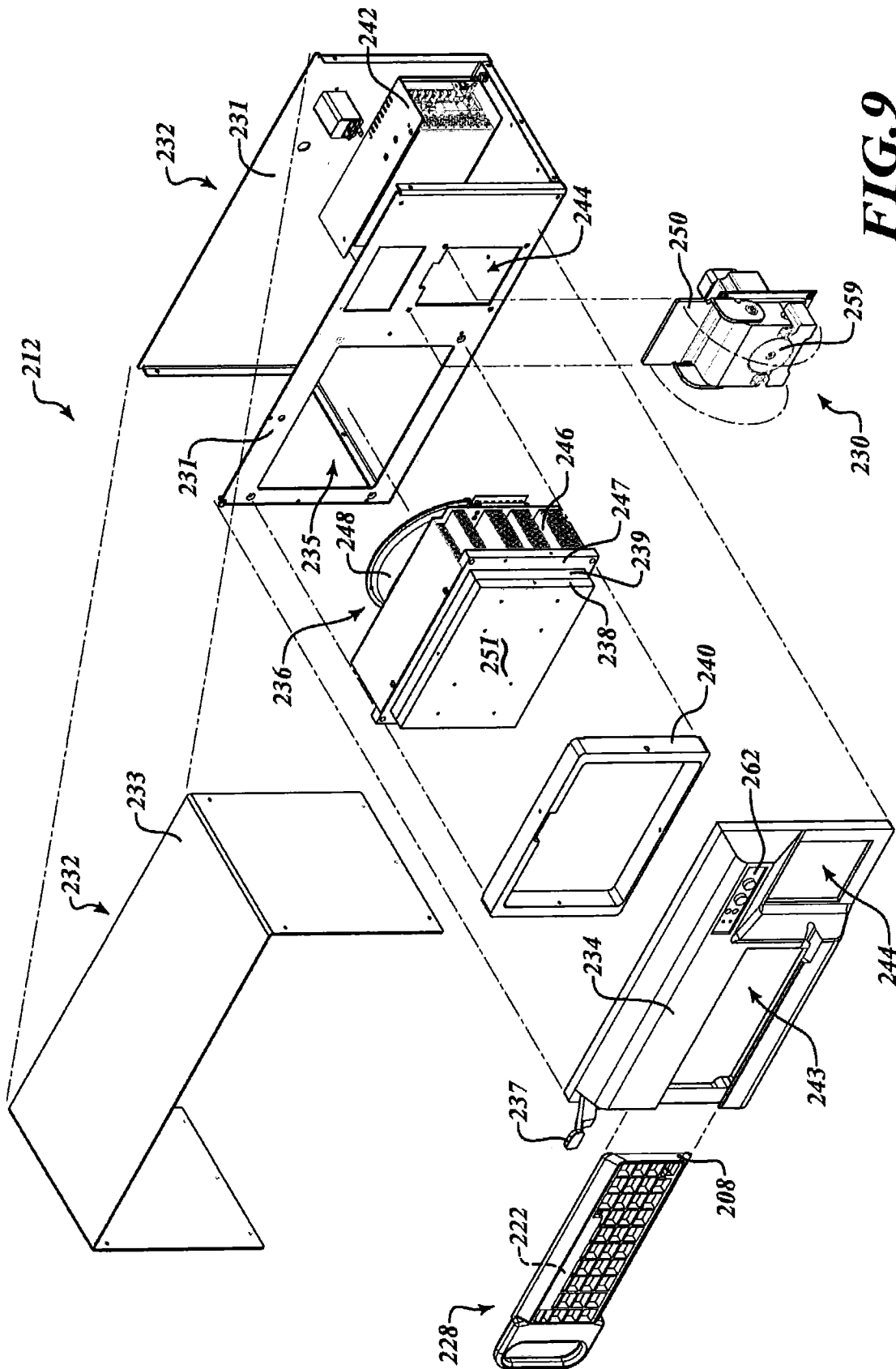


FIG. 9

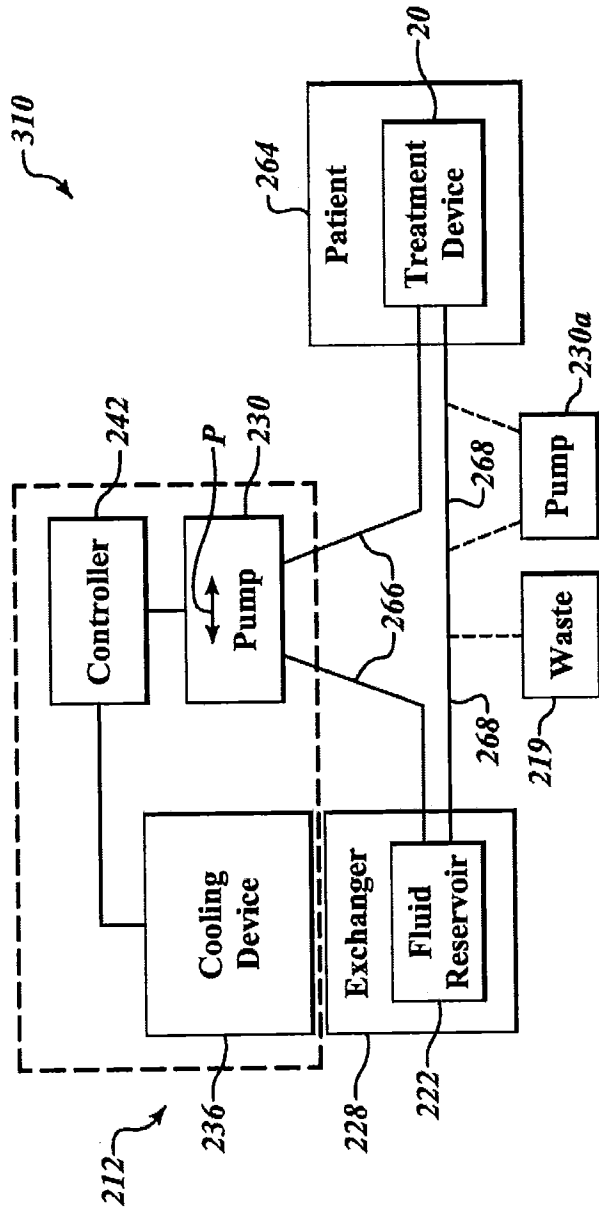


FIG. 10

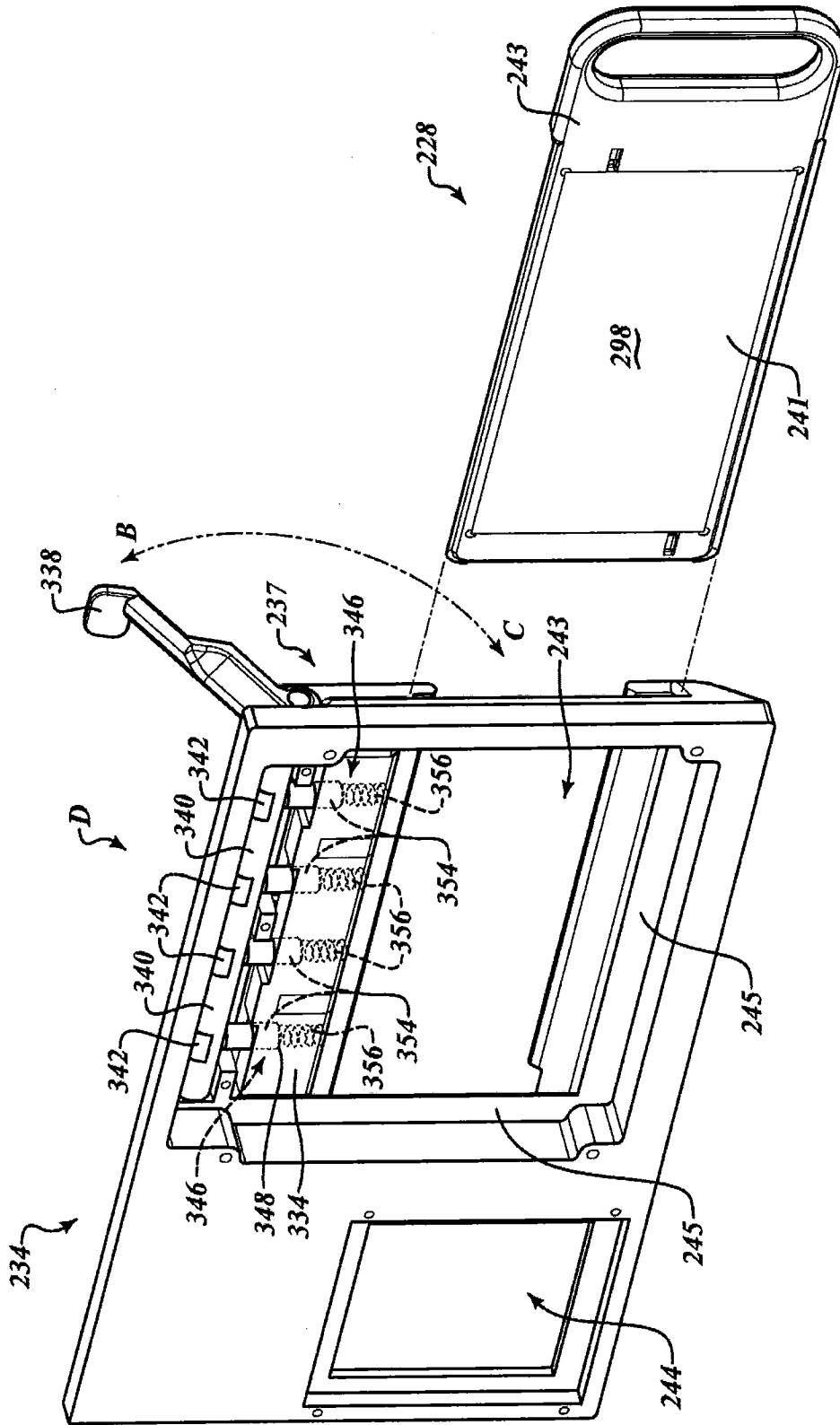


FIG. 11A

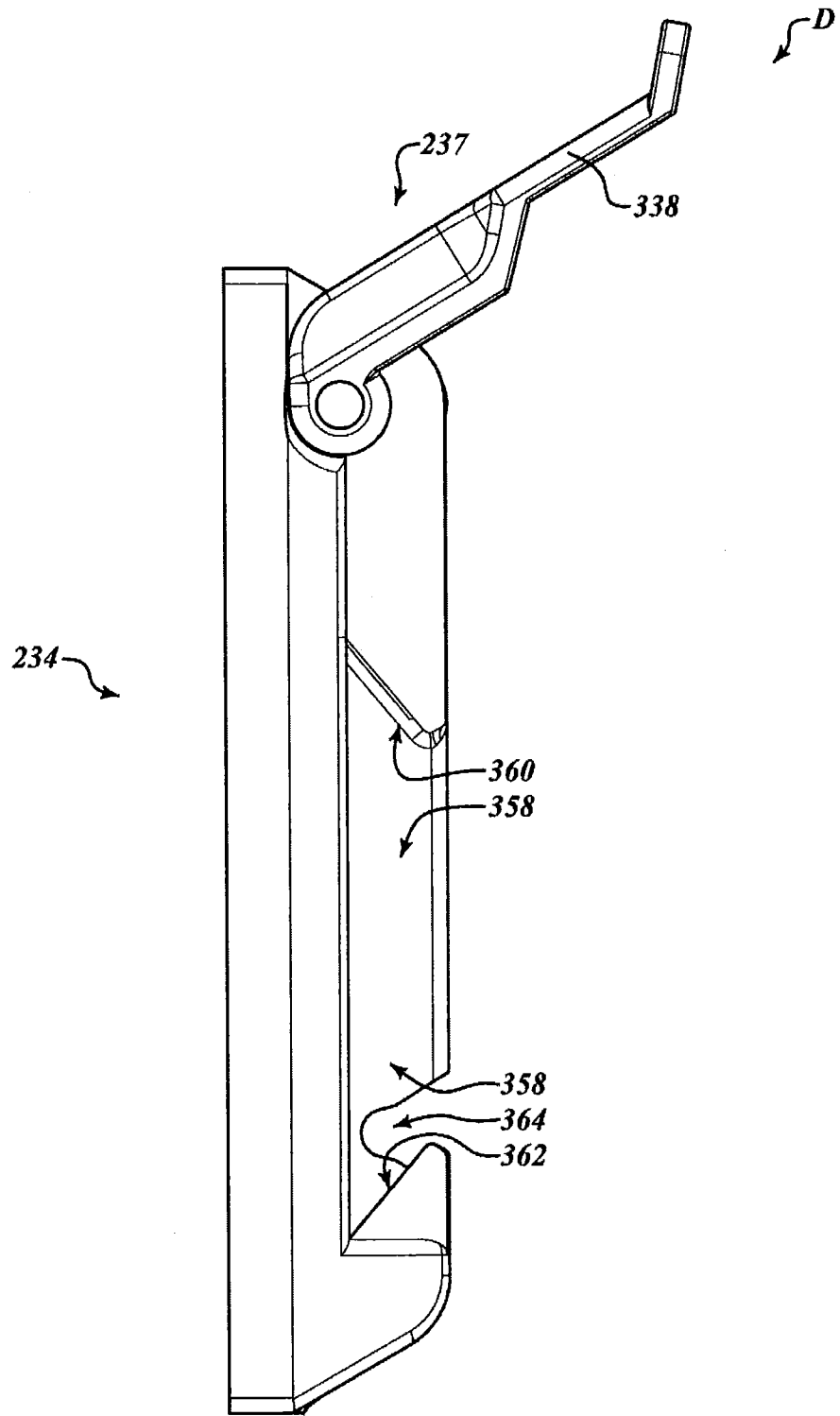


FIG. 12A

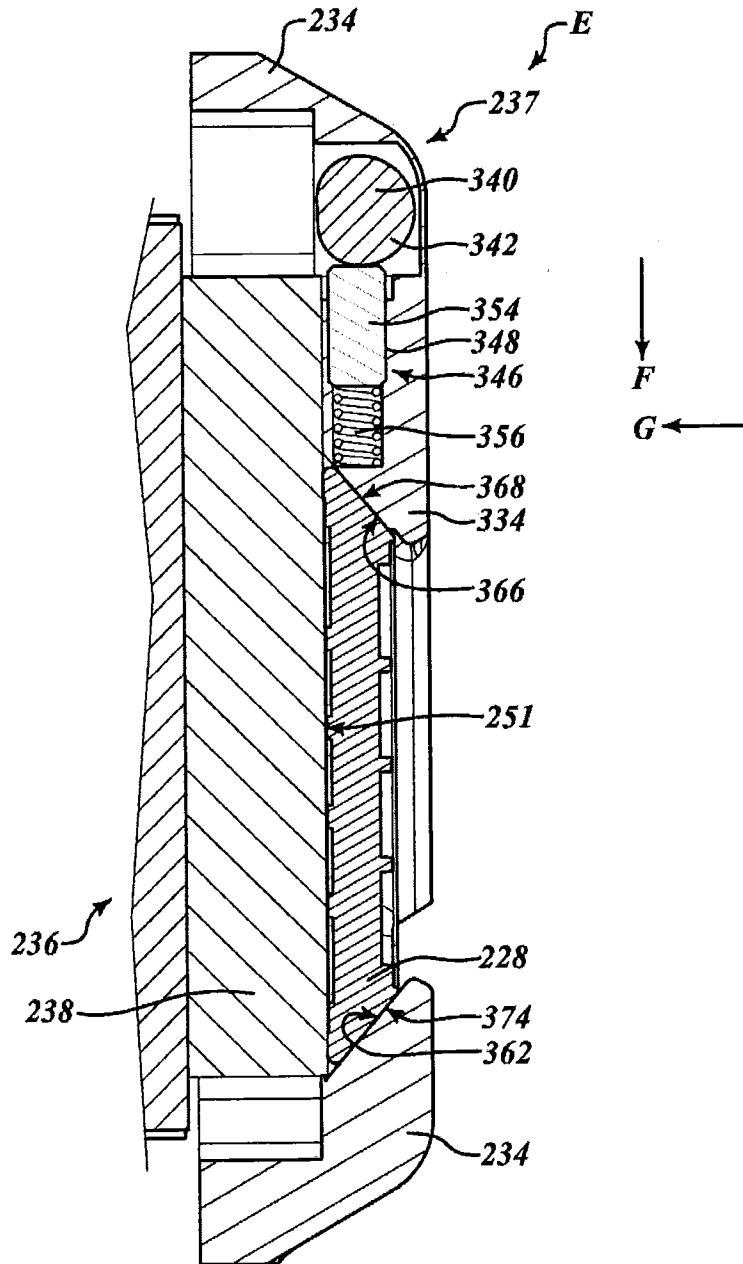


FIG. 12B

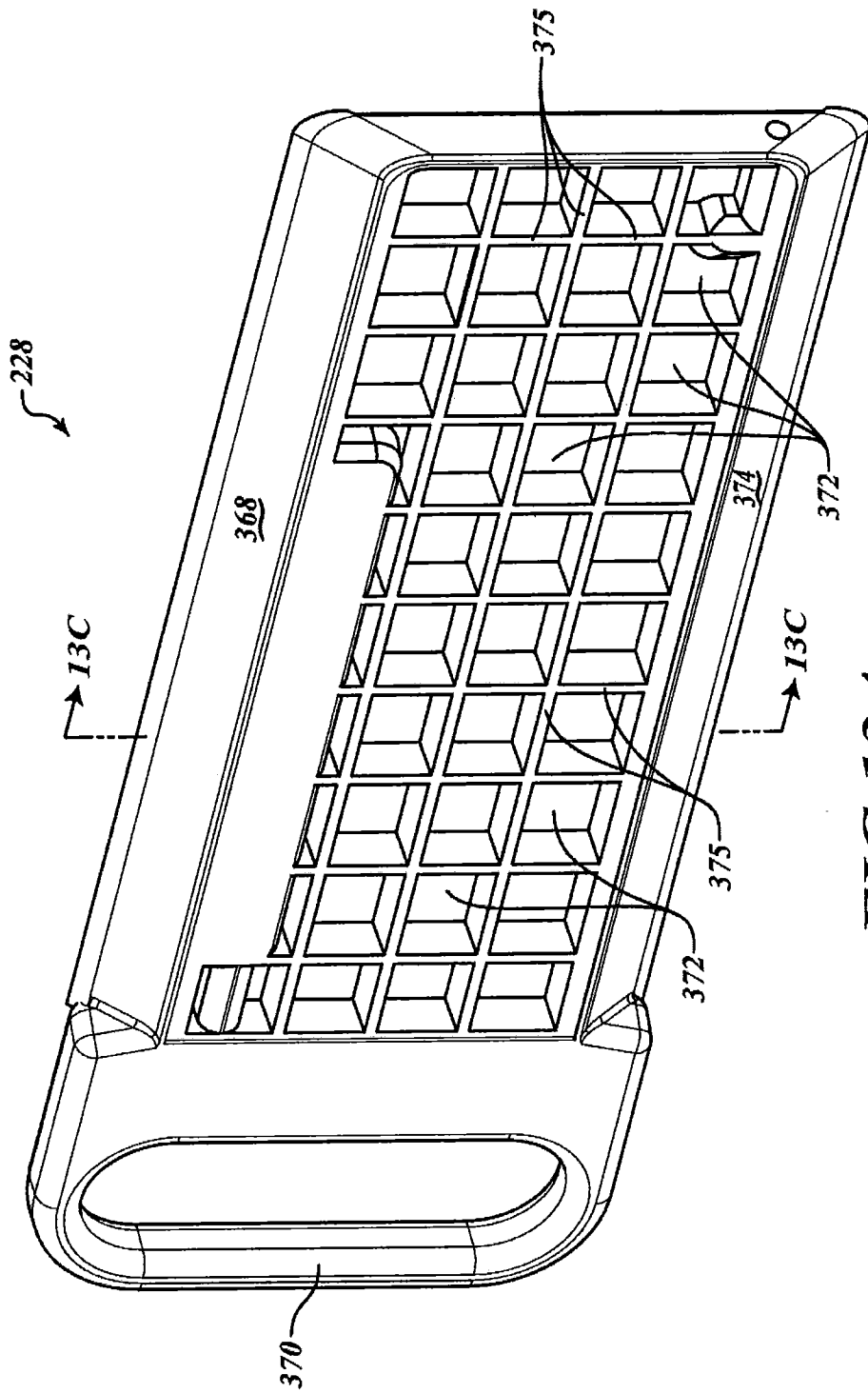


FIG. 13A

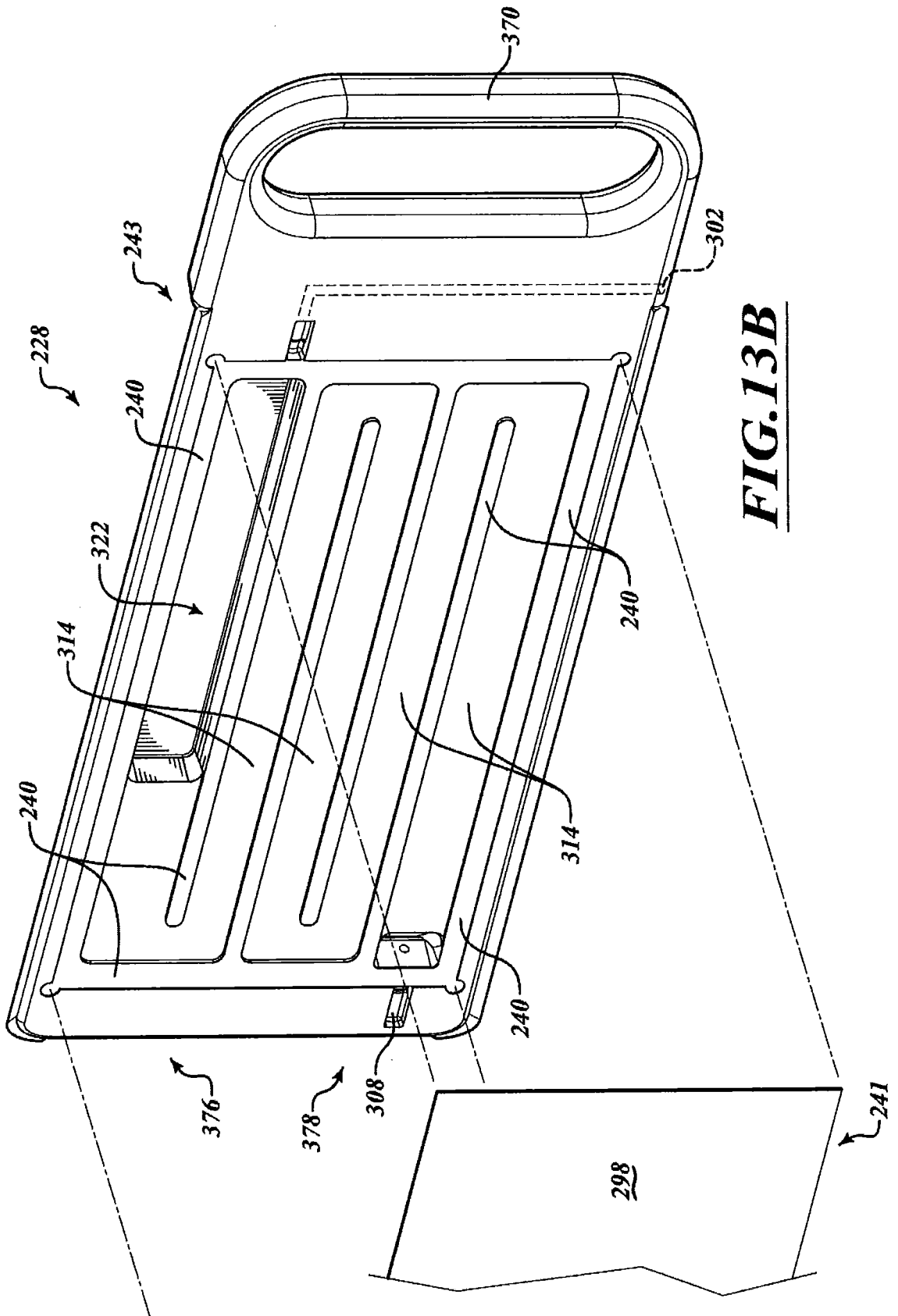


FIG. 13B

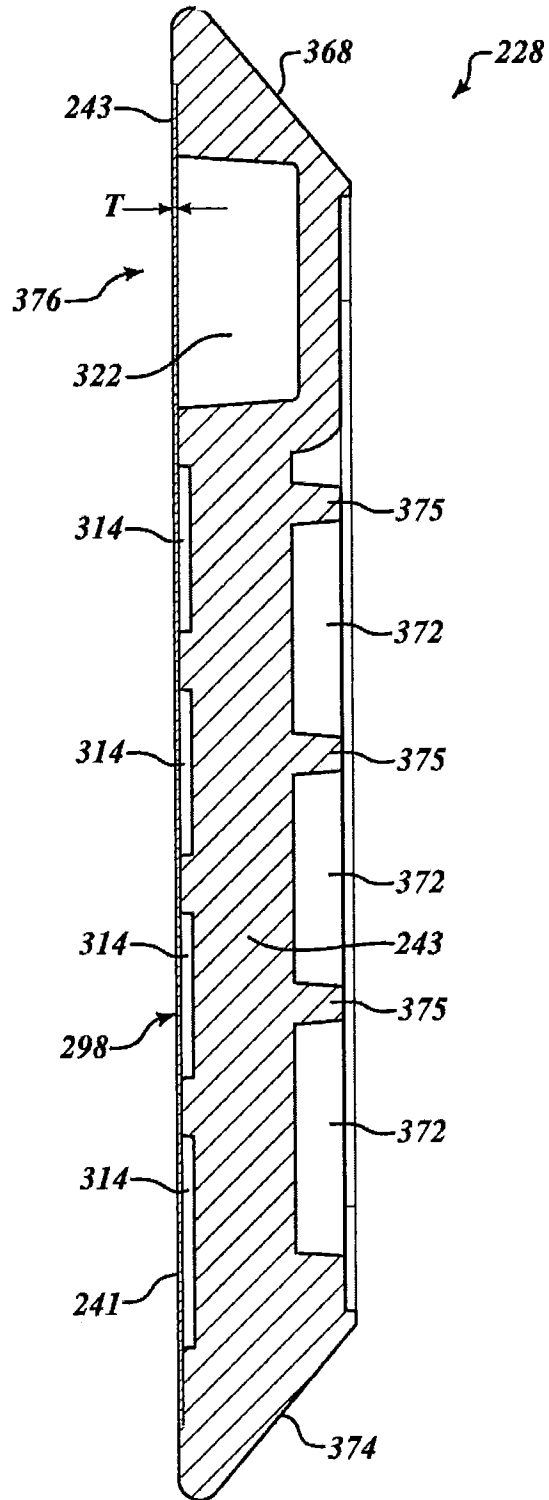


FIG. 13C

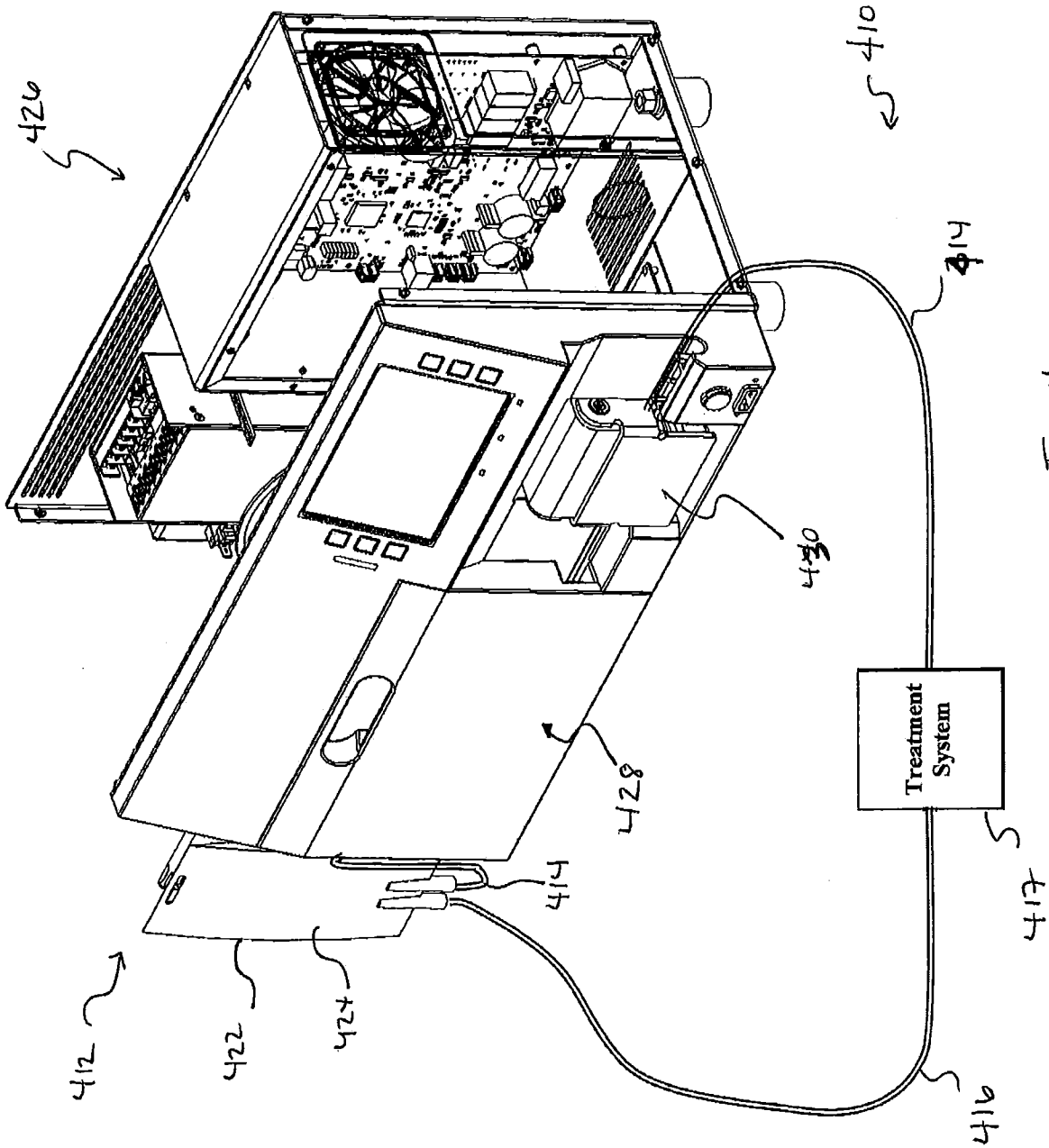


FIG.14

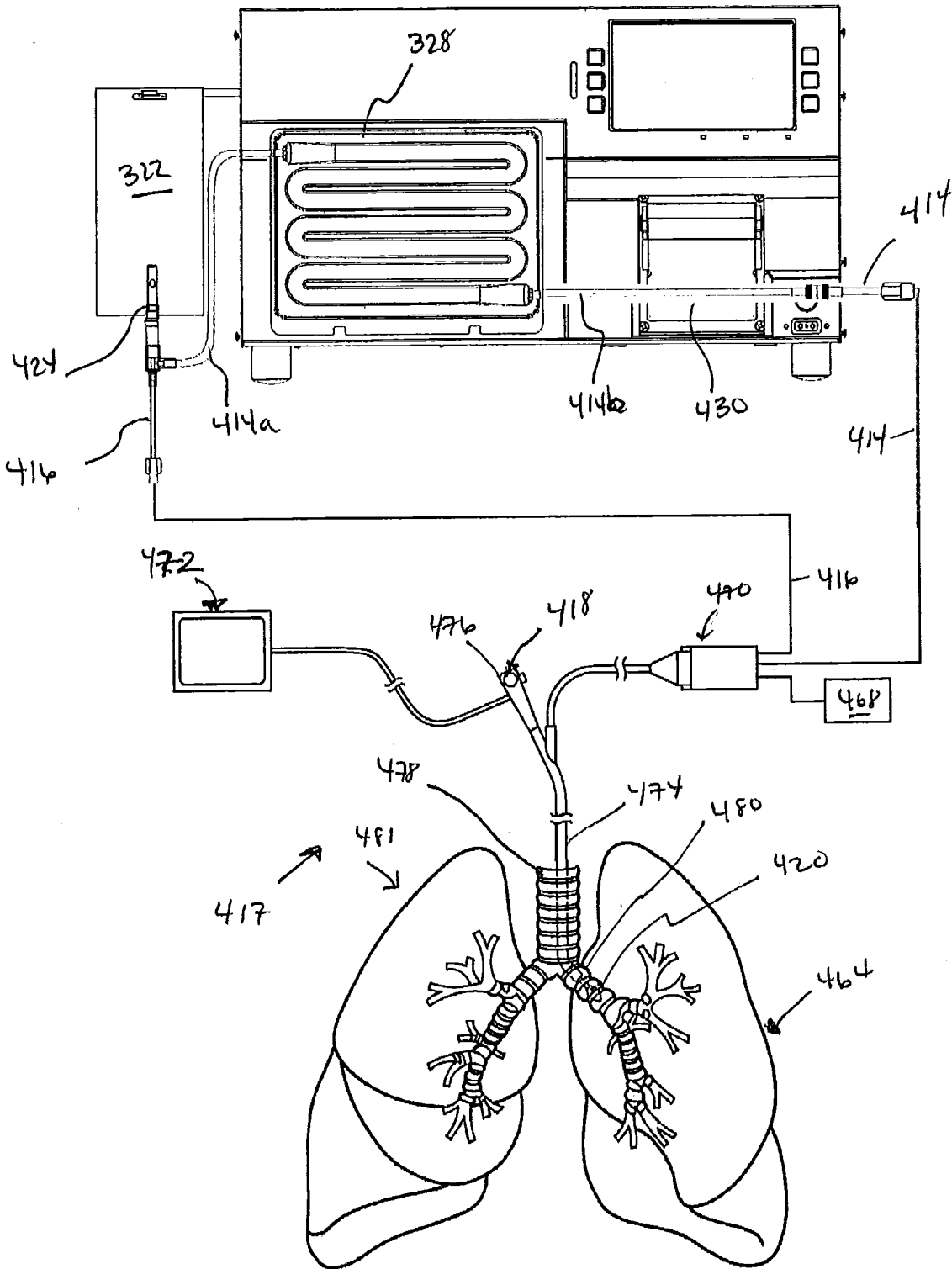


FIG. 15

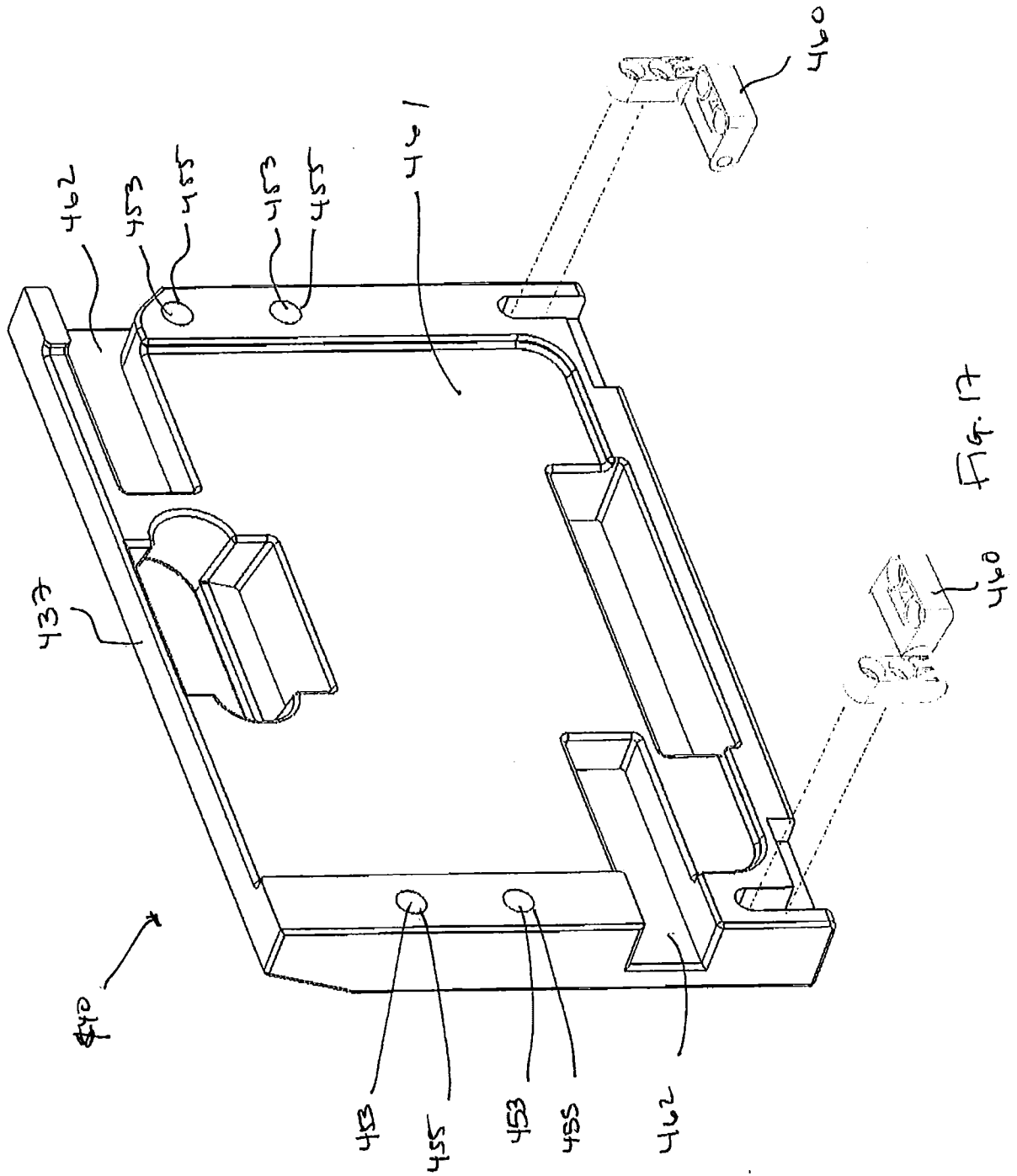


Fig. 17

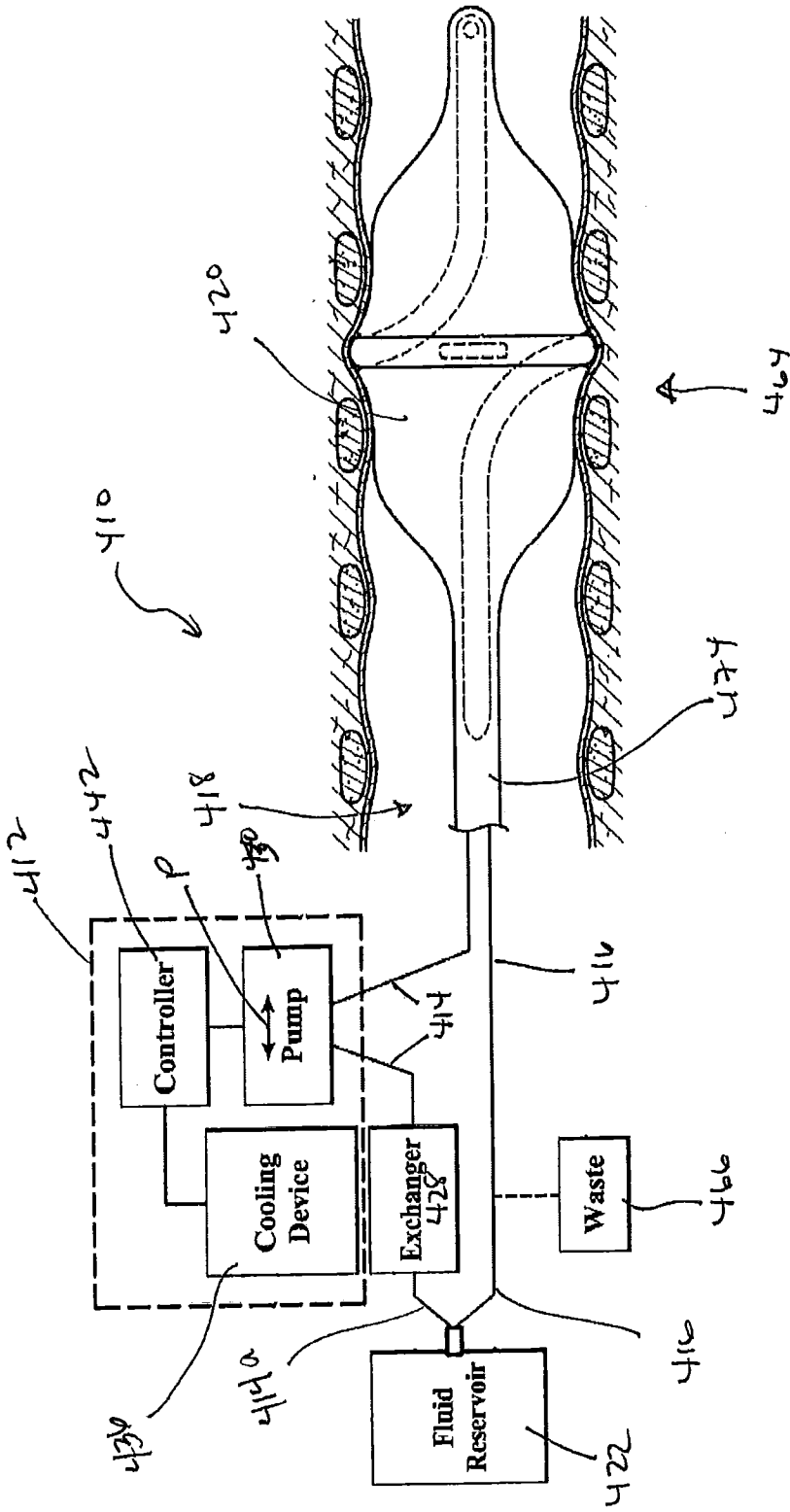


Fig. 18

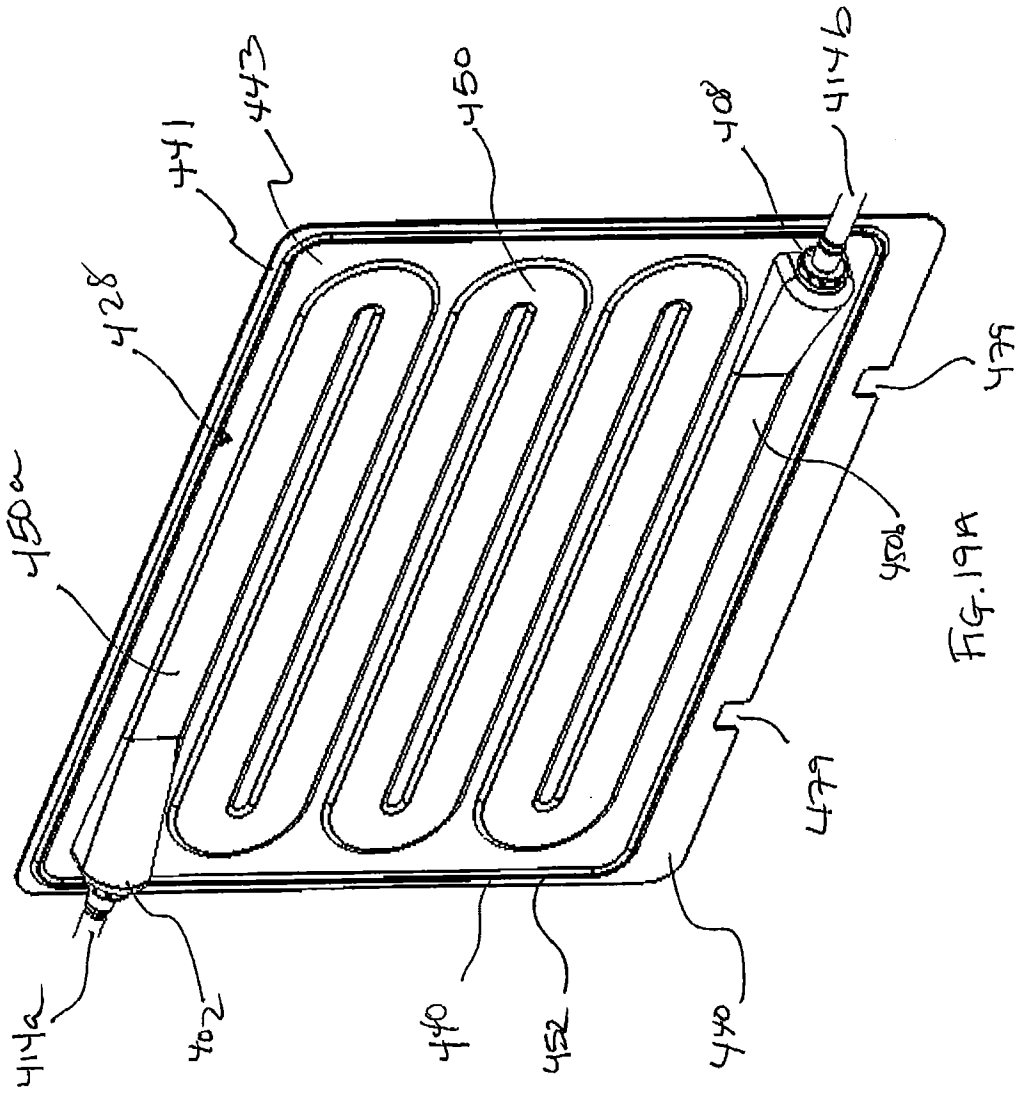


FIG. 19A

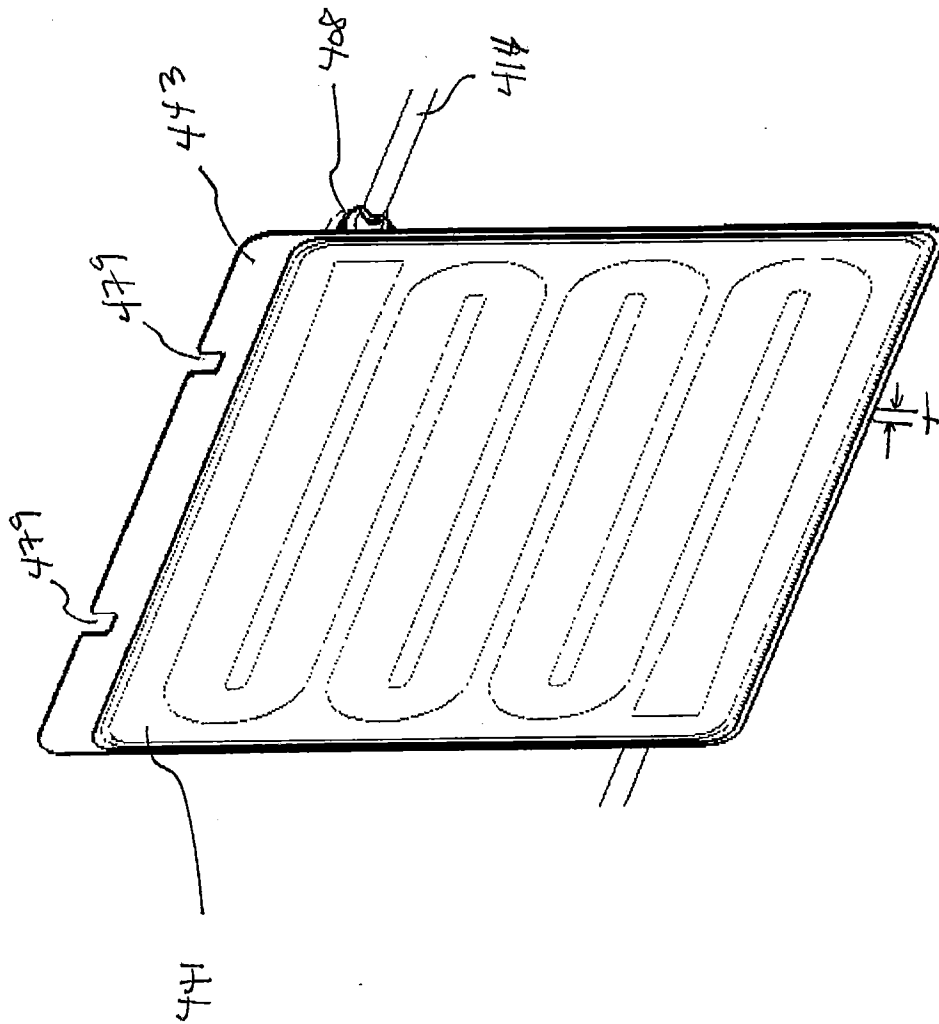


FIG. 19B

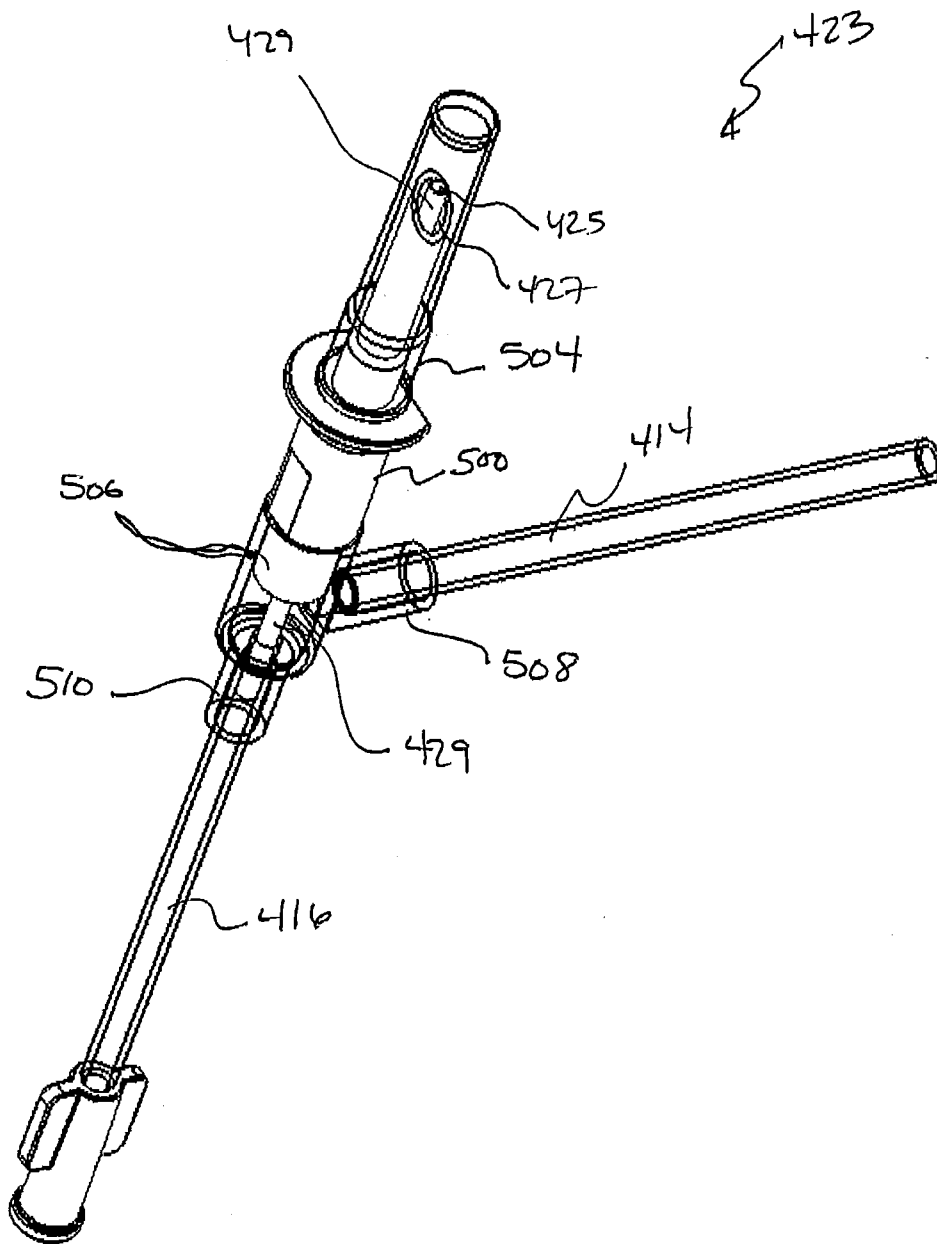


Fig. 20

INTERNATIONAL SEARCH REPORT

International application No.
PCT/US2014/026547**A. CLASSIFICATION OF SUBJECT MATTER****A61M 5/44(2006.01)i, A61M 5/14(2006.01)i**

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

A61M 5/44; A61B 18/14; A61M 31/00; A61M 25/10; A61F 7/12; A61M 1/00; A61B 18/18; A61F 7/00; A61M 5/14

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Korean utility models and applications for utility models

Japanese utility models and applications for utility models

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)

eKOMPASS(KIPO internal) & Keywords: Cooling device, fluid, heat exchanger, biasing mechanism, cartridge, pump, resilient body, thermoformed tray, bag strike assembly

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	US 2012-0095536 A1 (MACHOLD, T. R. et al.) 19 April 2012 See abstract; paragraphs [0002], [0016], [0017], [0082]-[0090], [0109]-[0114], [0127]-[0132], [0139]-[0142], [0154], [0186], [0179]; and figures 1, 2, 5, 6, 9-11.	1-14, 34, 45-52, 56, 58, 59, 61, 64-73, 76-84, 88-94, 96-101
A		15-17, 35-44, 53-55, 57, 60, 62, 63, 74, 75, 85-87, 95, 102-105, 119-169
A	US 2011-0301587 A1 (DEEM, M. et al.) 08 December 2011 See abstract; paragraphs [0013]-[0027], [0126]-[0133]; and figure 10.	1-17, 34-105, 119-169
A	US 6719779 B2 (DAOUD, A. G.) 13 April 2004 See abstract; columns 9-11; and figures 3, 4.	1-17, 34-105, 119-169
A	US 2008-0146995 A1 (SMISSON, H. F. et al.) 19 June 2008 See abstract; claims 1, 35; paragraphs [0016]-[0018]; and figures 1, 2.	1-17, 34-105, 119-169
A	US 2012-0283562 A2 (GINSBURG, R. et al.) 08 November 2012 See abstract; claims 1, 12, 13; paragraphs [0132]; and figure 16.	1-17, 34-105, 119-169

 Further documents are listed in the continuation of Box C. See patent family annex.

* Special categories of cited documents:

"A" document defining the general state of the art which is not considered to be of particular relevance

"E" earlier application or patent but published on or after the international filing date

"L" document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)

"O" document referring to an oral disclosure, use, exhibition or other means

"P" document published prior to the international filing date but later than the priority date claimed

"T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention

"X" document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone

"Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art

"&" document member of the same patent family

Date of the actual completion of the international search

15 July 2014 (15.07.2014)

Date of mailing of the international search report

15 July 2014 (15.07.2014)

Name and mailing address of the ISA/KR

International Application Division
Korean Intellectual Property Office
189 Cheongsa-ro, Seo-gu, Daejeon Metropolitan City, 302-701,
Republic of Korea

Facsimile No. +82-42-472-7140

Authorized officer

AHN, Jeong Hwan

Telephone No. +82-42-481-8440



INTERNATIONAL SEARCH REPORTInternational application No.
PCT/US2014/026547

C (Continuation). DOCUMENTS CONSIDERED TO BE RELEVANT		
Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
A	US 2010-0280454 A1 (ROSIELLO, K. M.) 04 November 2010 See abstract; claims 1,6; paragraphs [0048],[0052]-[0054]; and figures 1,2,5A.	1-17, 34-105 , 119-169

Box No. II Observations where certain claims were found unsearchable (Continuation of item 2 of first sheet)

This international search report has not been established in respect of certain claims under Article 17(2)(a) for the following reasons:

1. Claims Nos.: 18-33, 106-118
because they relate to subject matter not required to be searched by this Authority, namely:
Claims 18-33 and 106-118 pertain to methods for treatment of the human and thus relate to a subject-matter which this International Searching Authority is not required, under Article 17(2)(a)(i) of the PCT and Rule 39.1(iv) of the Regulations under the PCT, to search.
2. Claims Nos.:
because they relate to parts of the international application that do not comply with the prescribed requirements to such an extent that no meaningful international search can be carried out, specifically:
3. Claims Nos.:
because they are dependent claims and are not drafted in accordance with the second and third sentences of Rule 6.4(a).

Box No. III Observations where unity of invention is lacking (Continuation of item 3 of first sheet)

This International Searching Authority found multiple inventions in this international application, as follows:

1. As all required additional search fees were timely paid by the applicant, this international search report covers all searchable claims.
2. As all searchable claims could be searched without effort justifying an additional fees, this Authority did not invite payment of any additional fees.
3. As only some of the required additional search fees were timely paid by the applicant, this international search report covers only those claims for which fees were paid, specifically claims Nos.:
4. No required additional search fees were timely paid by the applicant. Consequently, this international search report is restricted to the invention first mentioned in the claims; it is covered by claims Nos.:

Remark on Protest

- The additional search fees were accompanied by the applicant's protest and, where applicable, the payment of a protest fee.
- The additional search fees were accompanied by the applicant's protest but the applicable protest fee was not paid within the time limit specified in the invitation.
- No protest accompanied the payment of additional search fees.

INTERNATIONAL SEARCH REPORT

Information on patent family members

International application No.

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