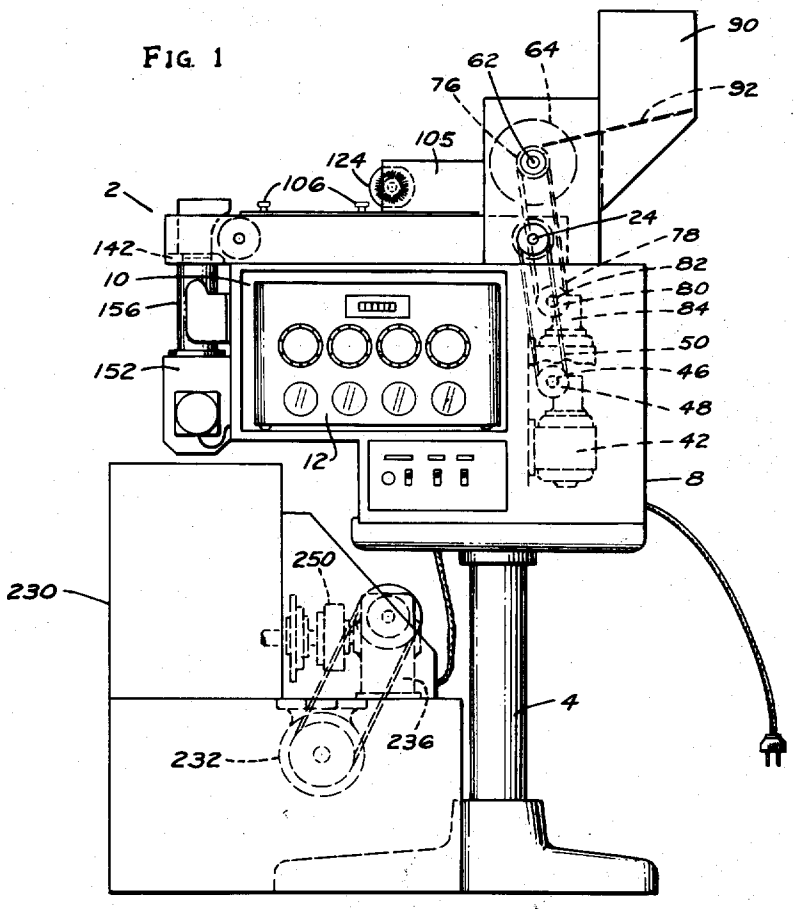
July 25, 1961 E. G. CLEVELAND ET AL Re. 25,013

APPARATUS FOR COUNTING AND PACKAGING ARTICLES

Original Filed Jan. 6, 1956 5 Sheets-Sheet 1

FIG. 1



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APPARATUS FOR COUNTING AND PACKAGING ARTICLES

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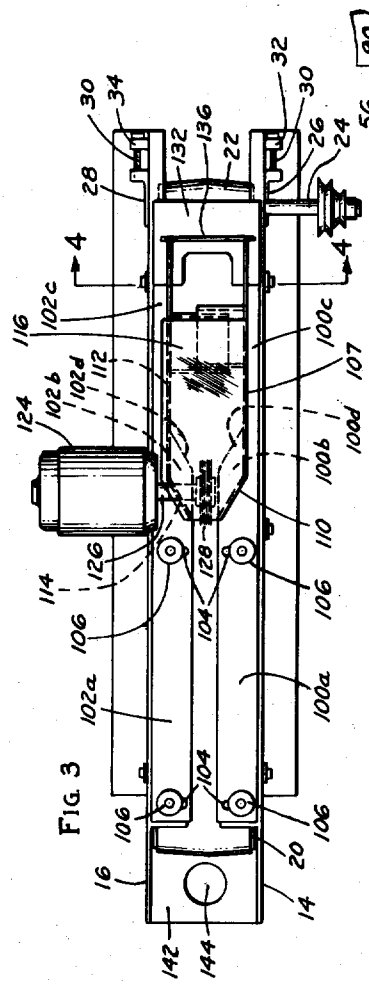


FIG. 3

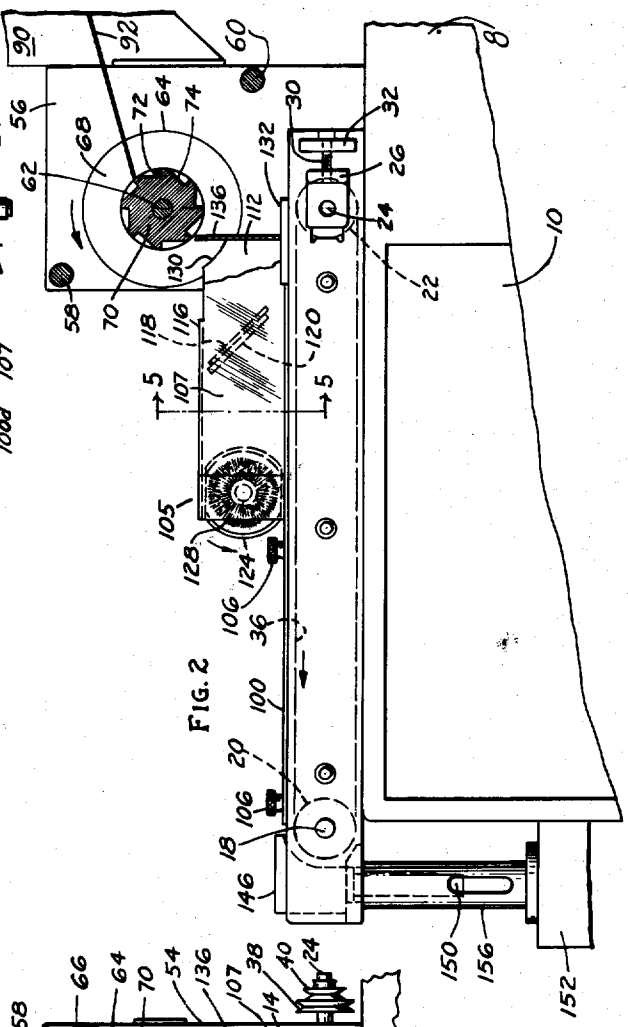


FIG. 2

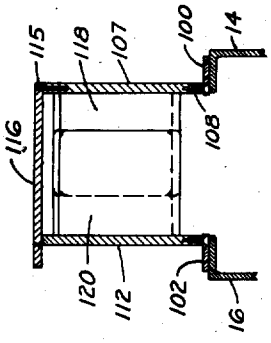


FIG. 5

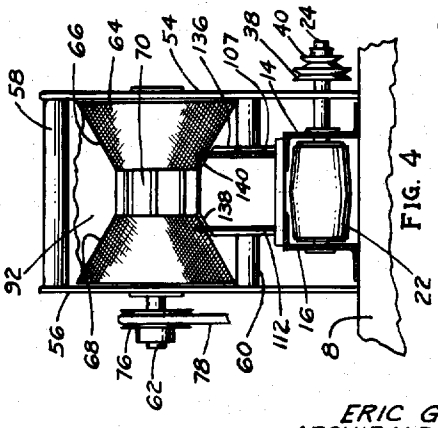


FIG. 4

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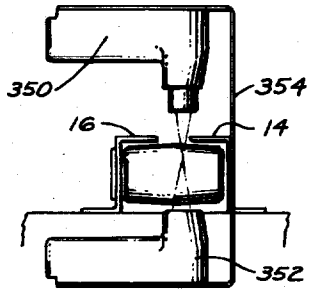


FIG. 8

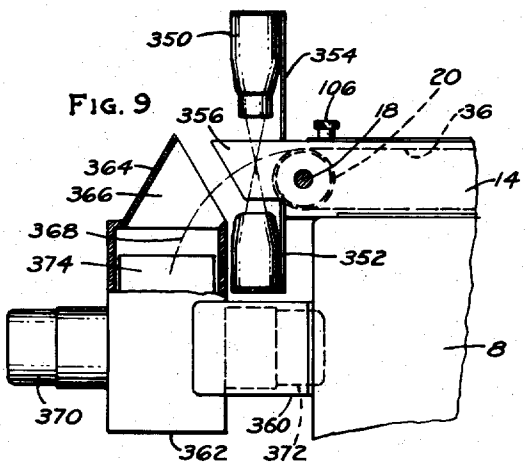


FIG. 9

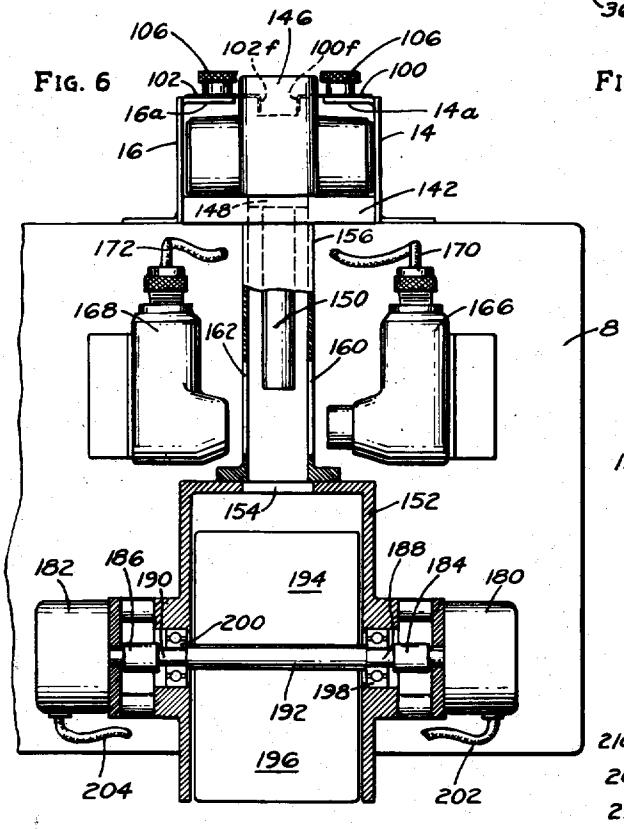


FIG. 6

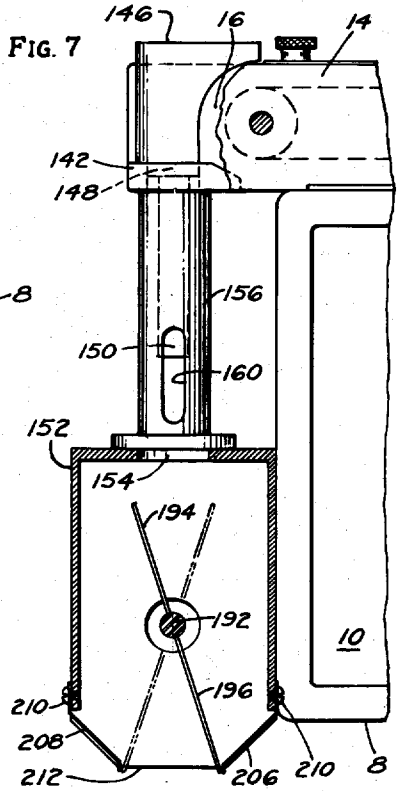


FIG. 7

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FIG. 10

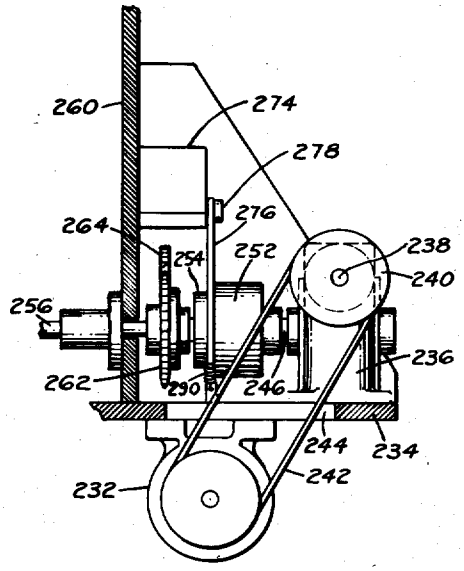
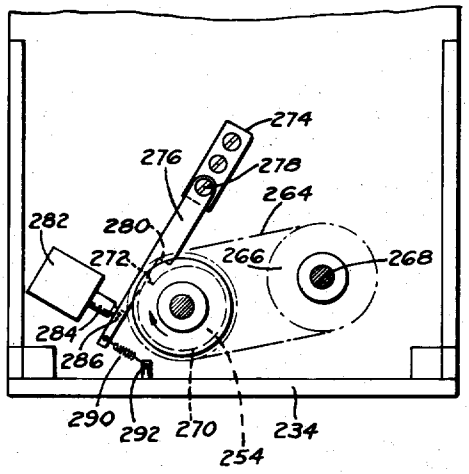


FIG. 11



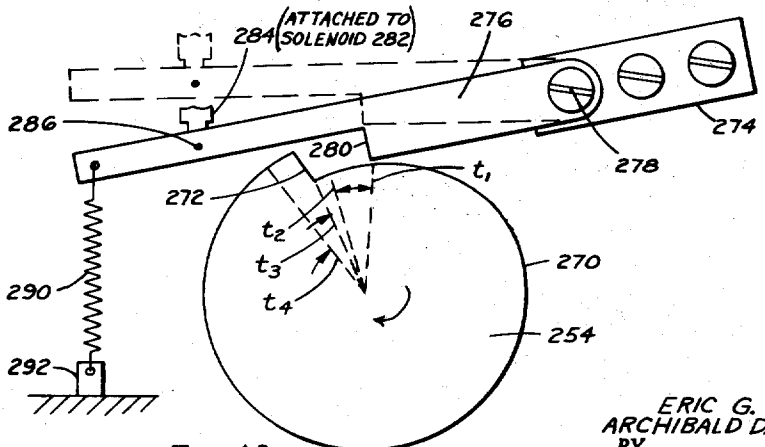


FIG. 13

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APPARATUS FOR COUNTING AND PACKAGING ARTICLES

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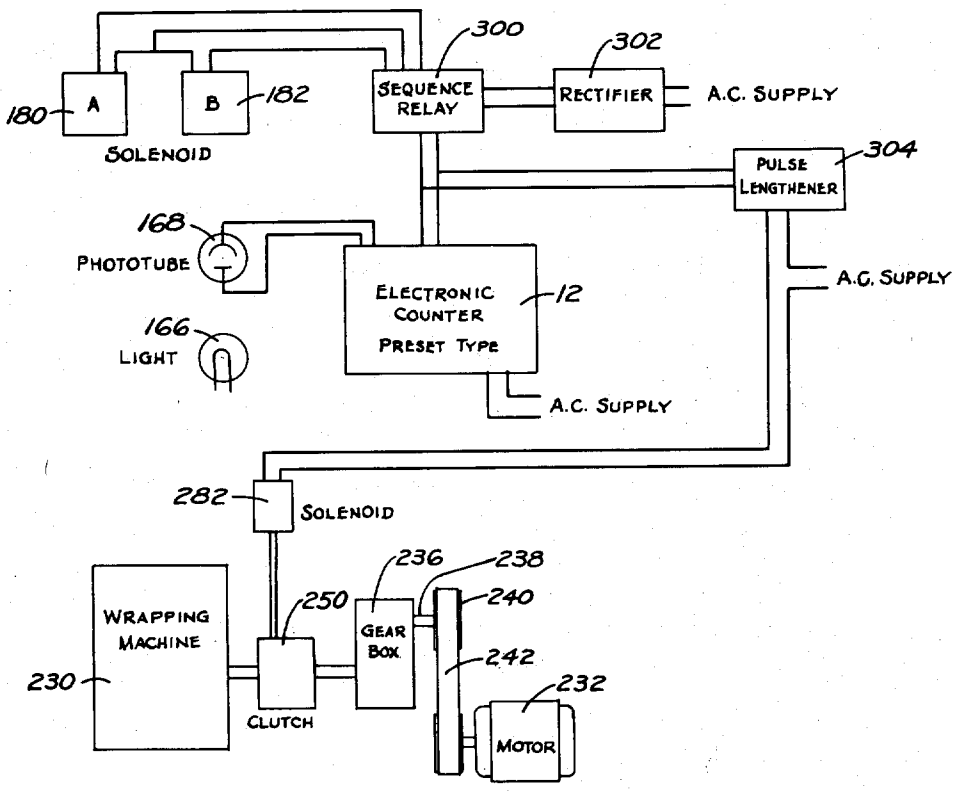


FIG. 12

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25,013

APPARATUS FOR COUNTING AND PACKAGING ARTICLES

Eric G. Cleveland, Melrose, and Archibald D. Standley, Jr., Danvers, Mass., assignors to Delta Engineering Corporation, Boston, Mass., a corporation of Massachusetts

Original No. 2,899,784, dated Aug. 18, 1959, Ser. No. 557,819, Jan. 6, 1956. Application for reissue Jan. 25, 1961, Ser. No. 90,596

11 Claims. (Cl. 53—61)

Matter enclosed in heavy brackets [ ] appears in the original patent but forms no part of this reissue specification; matter printed in italics indicates the additions made by reissue.

This invention relates to the art of counting and packaging articles. Heretofore various types of machines have been devised for counting articles, but such machines have generally suffered from excessive cost, mechanical failure, lack of speed, inability to count articles of various sizes, failure to render a dependable count under all conditions of operation, or lack of adaptability for use with other apparatus such as packaging or bottling machines.

Accordingly it is an object of this invention to provide a counting machine adapted to overcome the aforesaid difficulties attendant to previous machines.

Another object of this invention is to provide a combination counting and packaging system which automatically and reliably operates to count and package small articles at a rapid rate without the articles being contacted by human hands.

Another object of the present invention is to provide a method of automatically counting and packaging small articles at a rapid rate without error.

Other objects and many of the attendant advantages of this invention will be readily appreciated as the same becomes better understood by reference to the following detailed description when considered in connection with the accompanying drawings, wherein:

FIG. 1 is a side view in elevation of a novel electromechanical system for counting and packaging small articles according to the present invention;

FIG. 2 is an enlarged view, partly in section, of a portion of the counting machine embodied in the apparatus of FIG. 1;

FIG. 3 is a plan view of the structure shown in FIG. 2;

FIG. 4 is a vertical section taken along line 4—4 of FIG. 3;

FIG. 5 is a vertical section taken along line 5—5 of FIG. 2;

FIG. 6 is an enlarged front elevation, partly in section, of the front end of the counting machine;

FIG. 7 is an enlarged side elevation of the front end of the counting machine;

FIG. 8 is a fragmentary front view in elevation of a modified form of the counting machine;

FIG. 9 is a side view in elevation, partly in section, of the modified form of counting machine shown in FIG. 8;

FIG. 10 is an enlarged side view, partly in section, of the [drawing] *driving* mechanism for the packaging machine shown in FIG. 1;

FIG. 11 is an enlarged [side] *rear* elevation, partly in section, of the clutch assembly embodied in the driving mechanism shown in FIG. [11] *10*; [and,]

FIG. 12 is a schematic design of the electrical unit connecting the counting and packaging machines so as to cause the two machines to operate in synchronism[.]; *and*

FIG. 13 is an enlarged schematic view of certain parts of the clutch assembly illustrating the need for variable control of the counter's output signals.

Referring now to FIG. 1, there is shown a counting machine, generally indicated at **2**, mounted on a suitable supporting pedestal base **4** in proximity to a wrapping or packaging machine **230**. The packaging machine is conventional, and, therefore, is shown schematically except for its driving mechanism which is shown in more detail to indicate the extent to which it has been modified for the purposes of this invention as described hereafter.

Any packaging machine of the type utilizing the method similar to the one outlined in United States patent to L. L. Salfisberg No. 2,141,318 can be used. In practice we have found a machine of the type shown and described in detail in United States patent to H. P. Lewis No. 2,626,494 to be particularly adaptable for the purposes of the present invention.

For comprehension of the present invention the packaging apparatus may be briefly described as comprising two molding rolls each provided with a series of depressions on its surface, means for rotating the rolls in opposite directions in synchronism, and means for feeding two webs of thermoplastic material to the nip region of the molding rolls where the latter pick up the webs and bring them together, under pressure, the web being sealed transversely, then longitudinally, and then transversely in continuous succession, and means for directing articles to be packaged between the webs so as to cause the articles to be carried in the depressions in the molding rolls as the latter rotate and seal the webs together.

In conventional machines of this type the sealing rolls rotate at an intermittently predetermined speed. However, in order to provide a counting and packaging system that is completely accurate, it has been determined that the certain speed must be variably controlled and, for this reason, the driving mechanism of the packaging machine is modified, as explained hereinafter.

Counting machine **2** comprises an enclosure **8** mounted on base **4**. Enclosure **8** has a rectangular cavity **10** within which is positioned a conventional counter **12** of the type that can be preset to count to a predetermined subtotal, to actuate associated mechanism upon attainment of the sub-total, and to add the sub-totals together to maintain a running summary. A decitron counter, manufactured by the Post Machinery Company of Beverly, Massachusetts, has been used successfully in the illustrated installation.

Mounted on top of enclosure **8** are two Z-section side members **14** and **16**. Journaled in the two side members adjacent their front ends is a shaft **18** on which is mounted a guide roll or pulley **20** which tapers slightly toward its ends. A second pulley **22** is carried by a drive shaft **24** journaled in the two movable bearing blocks **26** and **28**. The bearing blocks, each carried in grooves in Z-members **14** and **16**, are movable toward and away from pulley **20** by means of screws **30** carried by collars **32** and **34** fixedly secured to the two Z-members **14** and **16**. Running about the two pulleys is an endless belt **36**. By moving pulley **22** it is possible to adjust the tension of belt **36**. Attached to the outer end of drive shaft **24** is a pair of pulleys **38** and **40** of different size. Mounted within casing **8** is a gear head electric motor **42** whose output shaft **46** rotates at a lesser speed than the motor's armature. Carried on one end of shaft **46** is a pulley **48** which acts through belt **50** to turn drive shaft **24**, pulley **22**, and belt **36**. It has been found that the machine operates successfully when the belt is driven at a rate of 120 feet per second.

Also secured to the top of enclosure **8** are two side plates **54** and **56** supported in parallel-spaced relation by a pair of braces **58** and **60**. Journaled in the two side

plates is a shaft **62** on which is fixedly carried a feed roll generally designated by numeral **64**. The shape of feed roll **64** conforms substantially to that of an hour glass, having tapered frusto-conical end sections tapering inwardly from its ends **66** and **68** and an intermediate constant diameter section **70** provided with a plurality of saw teeth whose radially extending faces **72** lead inclined step sections **74** in the direction of rotation of the feed roll. Carried on the outer end of shaft **62** is a pulley **76** which is driven by a belt **78** from a pulley **80** attached to output shaft **82** of a second gear-head motor **84** mounted within enclosure **8**.

Secured to side plates **54** and **56** at the rear edges is a hopper enclosure **90** having a sloping delivery member **92** whose forward edge terminates close to feed roll **64**. The forward end of delivery member **92** is shaped similarly to the upper end of vertical member **136** (described hereinafter) to nest close to the sections **66**, **68** and **70** of the feed roll.

Resting on top of **Z** members **14** and **16** are two guide members **100** and **102** each provided with elongated slots **104**. Thumb screws **106** extend through slots **104** and are screwed into threaded openings in the top flanges **14a** and **16a** of the two **Z** members. The two guide members **100** and **102**, run rearwardly from about the axis of pulley **20** to a point short of pulley **22**, and comprise three distinct sections indicated by numerals **100a**, b, c, and **102a**, b and c. Sections **100a** and **102a** are of greatest width and are flat throughout. Sections **100c** and **102c** are of narrowest width and are also flat. Section **100b** is intermediate sections **100a** and **100c**. Section **102b** bears a similar relation to sections **102a** and **102c**. Section **100b** is set at a compound angle to sections **100a** and **100c**, sloping downwardly (a) toward the center line of belt **36**, and (b) toward pulley **22**, terminating in knife edge **100d** positioned close to but preferably not touching belt **36**. Section **102b** slopes similarly toward the center of belt **36** and toward pulley **22**. Both guides **100** and **102** have depending flanges **100f** and **102f** respectively that terminate just short of engagement with belt **36**. Both flanges **100f** and **102f** extend rearwardly up to sloping sections **100b** and **102b**.

Secured to the top of the guide members is an enclosure generally designated as **105** (see FIGS. 1–5) adapted to vary in width as the guide members are moved toward and away from each other. Enclosure **105** comprises a first vertical side wall **107** attached to guide member **100** by means of screws **108** (FIG. 5). Side wall **107** includes at its front end an inwardly extending section **110** which terminates at the edge of guide section **100a**. A second side wall **112** similar to side wall **107** is secured to guide member **102**. Side wall **112** also has an inwardly extending section **114** similar to side wall **107**. Attached to the top edge of side wall **107** by means of pins **115** is a horizontal top wall **116** which lies in close sliding engagement with the top edge of side wall **112**. Secured to side walls **107** and **112** are two baffle members **18** and **120** which overlap each other and are inclined downwardly and rearwardly. Mounted on top of enclosure **8** is a third motor **124** having a drive shaft **126** that extends through the forward section **114** of side wall **112**. On the end of shaft **126** is a brush element **128** whose bristles are of such length as to extend substantially to the level of the top surface of guide members **100** and **102**. Motor **124** rotates brush **128** in a counterclockwise direction as viewed in FIG. 2. The rear edges of side walls **107** and **112** extend proximate to the feed roll and are cut away as at **130** (FIG. 2) to lie closely against the tapered parts of the feed roll. Secured to the two **Z** members **14** and **16** below the feed roll is a flat plate **132** which carries a vertical member **136** whose top corners are cut away as at **138** and **140** so as to fit close to the feed roll (see FIG. 4). Vertical member **136** abuts the rear ends of side wall members **107** and **112** and forms part of the enclosure **105**.

Mounted between **Z** members **14** and **16** at the front ends is a rectangular block **142** having a central opening **144**. Removably supported by block **142** is a curved guard member **146** undercut at its bottom to provide a circular extension **148** that fits within opening **144**. Formed integral with extension **148** is a vertical tube **150** of predetermined diameter.

Attached to the front wall of enclosure **8** is a small substantially rectangular casing **152** having an aperture **154** in its top. Secured to the top wall is a large tubular element **156** whose interior diameter is substantially the same as that of aperture **154**.

Tubular element **156** surrounds tube **150** and engages the bottom of block **142**. A pair of slits **160** and **162** are provided in tube **156** in diametrical opposite relation to each other. Secured to the front wall of casing **8** is a photoelectric eye assembly comprising a light source **166** and a phototube receiver **168**. Wires **170** and **172** (FIG. 6) extend through openings in the front wall of enclosure **8** and connect the photoelectric eye assembly to the electronic counter, as hereinafter explained.

A pair of rotary solenoids **180** and **182** are mounted at opposite sides of casing **152**. As seen in FIG. 6, the solenoids have output shafts coupled by suitable connecting means **184** and **186** to the ends **188** and **190** of a common shaft **192** carrying two radially extending blades **194** and **196** mounted in diametrical relation. The combination of shaft **192** and blades **194** and **196** may be termed a shutter or gate. Shaft **192** and blades **194** and **196** together with casing **152** function as a flow divider, as hereinafter explained. The ends **188** and **190** of shaft **192** are supported by bearings **198** and **200** mounted in openings in casing **152**. Cables **202** and **204** extend into casing **8** to provide electrical connections with the electric eye system and the electronic counter. The solenoids are identical in operation and, because they are oppositely mounted, act to turn shaft **192** in opposite directions. The open bottom end of casing **152** is partially blocked by two inclined barriers **206** and **208** (FIG. 7) which are removably secured to the casing by screws **210**. The two barriers define a narrow outlet **212**. Blade **196** extends into the outlet **212** and, depending upon which solenoid is energized, engages or extends close to the bottom edges of one of the barriers **206** and **208** as shown in FIG. 7. The solenoids rotate shaft **192** through limited arcs as, for example, **350**.

Assuming hopper **90** to be filled with articles to be counted, and the various motors and electrical components to be connected to suitable power supplies, operation of the above described counter unit is as follows:

The articles placed in hopper **90** fall by gravity along chute **92** toward the hour glass shaped feed roll **64**. The intermediate section **70** of feed roll **64**, rotating in a counterclockwise direction as viewed in FIG. 2, picks up the articles from inclined chute **92** and carries them around until they fall by gravity into enclosure **105**. Enclosure **105**, as seen in FIGS. 2 and 3, is open at the top adjacent the feed roll so as to receive the articles falling therefrom. The falling articles drop upon moving belt **36** and are carried thereby in the direction of guard member **146**. The articles fall upon belt **36** from the feed roll in a haphazard manner. Some of the articles may fall on deflecting members **118** and **120** which direct them downward onto the belt in front of base plate **132**. As the articles move forward on belt **36** the side walls **107** and **112** of enclosure **105** prevent them from overflowing the sides of the machine, and also assist sloping sections **100b** and **102b** in diverting them toward the center of the belt. Sloping sections **100b** and **102b** tend to cause the articles as they approach the forward end of enclosure **105** to aline themselves in a single file. This tendency is promoted by rotating brush **128** which, as seen in FIG. 2, rotates in a counterclockwise direction, and also by the guide members **100** and **102** which are adjusted so that the spacing between the

inner edges of sections 100a and 102a is just large enough to accommodate articles in single file, but not abreast of each other. The bristles of brush member 128 do not extend to any substantial degree below the upper surface of guide sections 100a and 102a, being of such length that the spacing between them and belt 36 is sufficient to allow passage of one article at a time. Thus, if one article is riding on another, the bristles of the brush will lightly engage the topmost article and toss it rearwardly in the direction of angularly positioned baffle members 118 and 120. Some of the articles that fall from sloping sections 100b and 102b fall against the sides of brush element 128 and are also thrown rearwardly. This prevents the articles from accumulating in the forward end of enclosure 105 and thereby tends to prevent more than one article at a time from passing into the channel formed by the two guide members. The forward sections 100a and 102a of the two guide members keep the articles alined in a single file as they are carried forward toward guard member 146 by belt 36. At the forward end the articles fall off of belt 36 into tube 150 carried by guard 146. Tube 150 is of such size as to prevent passage therethrough of more than one article at a time. As the articles pass out of the bottom end of tube 150 they interrupt the light beam emanating from light source 166. When this occurs phototube 168 sends a signal to counter 12, as explained hereinafter. The articles after interrupting the light beam pass through opening 154 into the casing 152. Assuming that barrier members 206 and 208 are secured to casing 152, the articles will collect at the bottom of the casing. The position of blades 194 and 196 of the flow divider determines on which side of the enclosure the articles will be directed. Thus if blade 96 lies close to or against barrier 206, as illustrated in FIG. 7, the articles will collect in the space between blade 96 and barrier 206. When the flow divider is turned to the dotted line position shown in FIG. 7, the articles will collect in the space between blade 96 and barrier 208. When solenoid 180 or 182, as the case may be, is energized, so as to reverse the position of blade 196, the articles that have collected in enclosure 152 will be discharged through opening 212 to the packaging machine, as explained hereinafter.

The reason for making barriers 206 and 208 detachable is so they can be removed from the counting unit when the latter is operated apart from the packaging machine. In the latter situation blades 194 and 196 serve merely to divert flow of articles from casing 152 from one direction to another as, for example, when one large container has been filled and it is desired to divert the articles into a second empty container. It is also to be understood that casing 152 and the various elements carried thereby may be entirely omitted when it is desired merely to provide a continuous count, as, for example, in determining inventory.

FIGS. 8 and 9 show a modified mounting for the electric eye sub-assembly of the counting unit. In this case a light source 350 and a phototube 352 are supported by a bracket 354 secured to an extension 356 of Z-member 14. Light source 350 and phototube 352 are so mounted that the light beam is directed vertically or substantially so through the path of articles falling from belt 36.

It is to be noted that the light beam must be properly positioned relative to the top of blade 194 in order to secure accurate and fast flow division. The proper distance between the light beam and the top of blade 194 is readily determined by trial and error.

Attached to casing 8 by means of a bracket 360 is a rectangular tube 362 provided at its open top end with an inclined front wall 364 and two triangular side walls. Only one side wall 366 is shown but it is to be understood that the missing wall is of the same shape and cooperates with front wall 364 and the other side wall 366 to provide a rectangular opening positioned to receive articles falling from conveyor 36. Dashed line 368 indicates the trajectory of articles falling from the conveyor. Tube 362 carries two rotary solenoids 370 and 372 corresponding to the solenoids 180 and 182. However, in this case the solenoids are mounted fore and aft rather than on the sides of the tube as are solenoids 180 and 182. Solenoids 370 and 372 operate a butterfly closure member 374 which controls the flow of articles through the tube. In this case the closure member engages opposite side walls of the tube alternately. However, if desired, the length of the blades may be shortened so as to merely divert and not to temporarily impede the flow of articles out of the bottom of tube 362.

FIGS. 1, 10 and 11 show a drive system for a packaging machine 230, the drive system being so constructed as to permit the packaging machine to be controlled by the above described counter unit.

A motor 232 is secured to the under side of a wall member 234 of the packaging machine. A gear reduction unit 236 is mounted on the upper side of the same wall member. The gear reduction unit has an input shaft 238 provided with a pulley 240 which is driven by motor 232 acting through belt 242 extending through an opening 244 in wall 234. The output shaft 246 of the gear reduction unit is connected to a single revolution clutch 250 having a first clutch member 252 affixed to shaft 246 and a second clutch member 254 in frictional engagement with the first member 252. Securely connected to second clutch member 254 is an output shaft 256 which is journaled in a vertical wall 260 of the packaging machine. Shaft 256 may be connected to a knife cutter for severing the web of packets produced by the packaging machine. Also carried by shaft 256 in fixed relation to clutch member 254 is a sprocket 262 carrying a drive chain 264 for driving the sealing rolls or dies of the packaging machine. FIG. 11 schematically shows chain 264 attached to a second sprocket 266 carried by a shaft 268. It is to be understood that shaft 268 is operatively coupled to drive two sealing rolls of the type embodied in the packaging devices illustrated in the aforementioned Salfisberg Patent No. 2,141,318 and Lewis Patent No. 2,626,494.

The exterior surface of clutch member 254 is machined to provide a cam surface 270 whose radius varies to provide an abutment 272. Mounted on the rear side of vertical wall 260 of the packaging machine is a bracket 274 which carries a lever 276 pivoted at 278. One edge of lever 276 is notched as at 280 to engage abutment 272 of the cam surface 270. A solenoid 282 (omitted in FIG. 10 for clarity but shown in FIG. 11) has its armature 284 pivotally connected at 286 to lever 276.

A tension spring 290 secured at 292 to wall member 234 normally keeps lever 276 in engagement with cam surface 270. So long as lever 276 is in locking engagement with abutment 272, second clutch member 254 and also sprocket 262 and shaft 256 remain at rest notwithstanding rotation of first clutch member 252. When solenoid 282 is energized (as explained hereinafter), armature 284 is withdrawn, causing lever 276 to rotate clockwise (FIG. 11) away from clutch member 254. This frees clutch member 254 and allows it to be driven by clutch member 252. Upon deenergization of the solenoid, spring 290 pulls lever 276 counterclockwise into engagement with cam surface 270. Shaft 256 and clutch member 254 continue to rotate until the abutment 272 is again engaged by the notched portion 280 of lever 276, whereupon driven clutch element 252 continues to rotate but driven clutch member 254 and shaft 256 come to rest.

FIG. 12 is a schematic circuit diagram including additional components not previously referred to. Although in this figure the connection between clutch 250 and wrapping machine 230 is shown only schematically, it is to be understood that actually the connection is as shown in FIG. 10. Moreover, although the various mo-

tors of the counting machine are not shown in the diagram, it is to be understood that they, and also the motors of the packaging machine, are all adapted to be coupled to an alternating current power supply. It is also to be understood that the circuit is the same when the electric eye system is mounted as in FIGS. 8 and 9.

In the circuit, electronic counter **12** is the primary or steering component. Any conventional electronic counter, other than the Post counter referred to previously, may be employed provided it is adapted to yield an output signal when a predetermined number of input signals have been applied to it. The counter should also be adapted to supply a visual indication of the total number of input signals applied to it, as well as being adjustable as to the number of input signals required to produce an output signal.

Counter **12** is connected to a suitable A.C. supply. Phototube **168** is connected to it, as indicated, so as to provide an input signal. Although not shown it is to be understood that light **166** is also connected to a suitable source of power. On its output side counter **12** is connected to a sequence relay **300**. The latter is connected to rectifier **302** on one side and to solenoids **180** and **182** on the other side. Sequence relay **300** functions as a switch to alternately connect solenoids **180** and **182** to rectifier **302**. Relay **300** is triggered by the output signals from counter **12**. Upon application of a signal from counter **12**, relay **300** changes its connection from one to the other of solenoids **180** and **182** and maintains its new connections until another signal is applied to it from counter **12**, whereupon it again changes its connections. In this way solenoids **180** and **182** are alternately energized by the output of rectifier **302**. Thus, for example, assuming solenoid **180** to be energized and solenoid **182** to be de-energized, when a pulse or signal is produced by counter **12** it is automatically applied to sequence relay **300**, causing the latter to break its circuit with solenoid **180** and to close the circuit with solenoid **182**. When this occurs solenoid **182** operates shaft **192** to turn blades **194** and **196** to a new position as shown in FIG. 7.

The output of counter **12** is also connected on its output side to a pulse lengthener **304** which operates to increase the time duration of the signal output from the counter. Pulse lengthener **304** is connected to clutch solenoid **282** so as to apply its output thereto. The purpose of pulse lengthener **304** will be explained hereinafter. However, it is to be noted that most electronic counters hitherto available are capable of producing output pulses only of fixed time duration. While some conventional counters may be adapted to selectively provide output pulses of various time durations, the output pulse periods are generally fixed so that it is difficult to vary the pulse periods as desired. Pulse lengthener **304** permits this variation over a relatively large range so that the duration of the pulses applied to solenoid **282** can be increased or decreased as required. Solenoid **282**, as explained previously, controls operation of clutch **250**.

The circuit operates as follows: Each time that an article falls down out of tube **150** it interrupts the beam of light from lamp **166**. When this occurs phototube **168** supplies an input signal to counter **12**. Counter **12** is preset to produce an output signal in response to a predetermined number of successive input signals, and when this occurs, the output signal is applied to relay **300** and pulse lengthener **304** simultaneously. Relay **300** functions as a switch to alternately connect solenoids **180** and **182** to rectifier **302**. When the output of counter **12** is applied to relay **300**, the latter automatically performs its switching function. Assuming one of the solenoids **180** and **182** to be energized from rectifier **302** acting through relay **300**, it remains energized until relay **300** is actuated by a signal from counter **12**. As explained previously, when one of the solenoids **180** and **182** is energized it operates to change the position of blades **194** and **196** and to keep said blades in that position until it is de-energized, whereupon the blades will be restored to their original position by the other newly energized solenoids.

Upon rotation of blades **194** and **196**, the articles retained in housing **152** are released through discharge opening **212** and fall into the packaging machine **230** between the webs of packaging material therein in the region where the packaging material comes together between the sealing rolls.

While this is occurring pulse lengthener **304** generates, in response to the output of counter **12**, an output pulse that is applied to and energizes solenoid **282**. When this occurs lever arm **276** is withdrawn from clutch element **254**, allowing the latter to be driven by clutch element **252** to drive the sealing rolls of the packaging machine. Solenoid **282** remains energized for the time duration of the pulse applied to it by pulse lengthener **304**. As soon as the pulse has terminated, solenoid **282** becomes de-energized and lever **276** is drawn by spring **290** back into engagement with the cam surface **270** of clutch element **254**. This occurs before the clutch completes one complete revolution so that when clutch element **254** returns to its original position abutment **280** will be engaged by lever **276** to stop rotation of clutch element **254**. As provided in the aforesaid Lewis patent, it is to be understood that the sealing rolls will advance or index one package for each revolution of the drive sprocket. Thus the single revolution clutch effectively causes the drive sprocket to make a single revolution, stop, make another revolution, stop, and so on, so as to cause the sealing rolls to advance one package at a time. Each time that a charge is released to the packaging machine clutch **250** will be actuated by counter **12** to cause operation of the sealing rolls.

The reason for including pulse lengthener **304** in the circuit is important. In general it was found that the output of commercially available counters was variable as to time duration to only a limited degree. Thus, for example, a particular type of counter would be adapted to give output signals having a time duration of $\frac{1}{10}$, $\frac{1}{4}$ and 1 second, as desired, but was incapable of having a pulse duration of any other desired amount between the upper and lower limits. Accordingly the counter and packaging machine had to be geared to provide synchronization using one of the available outputs of the counter. Although this was possible it limited output of the system because of the need for so timing the operation of the clutch solenoid as to prevent two charges from being deposited in the same package formed by the packaging machine.

FIG. 13 illustrates how such an error could arise and how the pulse lengthener acts to prevent such an error. In this figure assume, for example, that clutch member **270** is rotating at such a rate that the angle subtended by dotted lines t, and t_4 represents a time interval of $\frac{1}{2}$ second, the angle subtended by lines t, and t_2 represents a time interval of $\frac{1}{4}$ second, and the angle subtended by lines t, and t_3 represents a time interval of $\frac{1}{3}$ second. Assume also that the abutment **272** and the lever **276** are instantaneously in the position illustrated, with the solenoid having previously been energized and de-energized in response to a signal from the counter, having a time duration of $\frac{1}{4}$ second. If now a new signal is applied to solenoid **282**, lever **276** would move to the position shown in phantom and then return to its original position, all in a time period of about $\frac{1}{4}$ second. However, since the clutch abutment was spaced away from its normal point of contact with lever **276** by $\frac{1}{3}$ of a second, upon termination of said new signal, the lever would still be engaged by the abutment and disengage the clutch. Since the time the new signal was applied to solenoid **282** a new charge was being introduced to the packaging machine, the new charge would be added to the previous charge which would not have moved out of position due to the clutch being again disengaged. However, if now

the time duration of the output pulse of the counter had been increased to ½ second by adjustment of the pulse lengthener, lever **276** would be held away from cam surface **270** until abutment **272** has passed beyond its normal point of contact with lever **276**. Thus lever **276** would re-engage cam surface **270** behind abutment **272** and would ride on the cam surface without causing disengagement of the clutch until abutment **272** came around again. Consequently, the packaging machine will continue to operate so as to separately package successive charges. It is to be noted that the speed at which a charge is wrapped when the wrapping machine is operating varies with the package size.

It is believed apparent, therefore, that the pulse lengthener is an important element of the circuit. However, it may be omitted provided a counter is used whose output is capable of being varied as desired relative to pulse period.

The need for the clutch and means for varying the length of the output pulse of the counter arises due to the fact that the charges are not deposited in the wrapping machine at a constant rate. If the rate at which charges are deposited in the wrapping machine were constant, then the machine could be set to run at a constant speed and the clutch could be eliminated. In the absence of a constant rate of feed the machine must be capable of operation intermittently. Hence the use of a clutch. However, under optimum conditions achieved by the present invention, the wrapping machine approximates a continuously operating device. Thus, for example, when the counter is processing articles at an average rate of 1100 per minute, and segregating them into charges or groups of 100, the rate at which charges are released to the wrapping machine is substantially or almost constant so that the speed of the wrapper can be set at a level such that the wrapper will operate substantially continuously. The clutch provides assurance that the wrapper will keep in step with variation in the charge rate and the pulse lengthener provides assurance against overlap so that successive charges will be separately packaged.

It is contemplated that a magnetic detector may be employed in place of the electric eye system. It is also appreciated that the output signal from the electronic counter may be used to control other equipment, e.g., when filling boxes, to energize box closing equipment.

One distinct advantage of the counting machine resides in the two adjustable guide members provided to channel the articles in single file. Because the guides are adjustable the machine is capable of successfully handling many different types of articles with little loss of time for making the machine ready for operation.

Obviously many modifications and variations of the present invention are possible in the light of the above teachings. Therefore, it is to be understood that the invention is not limited in its application to the details of construction and arrangement of parts specifically described or illustrated, and that within the scope of the appended claims it may be practiced otherwise than as specifically described or illustrated.

We claim:

1. Apparatus for counting articles and packaging said articles in predetermined quantities comprising, means for storing a plurality of like articles, means for removing said articles from said storage means, means for separating said removed articles and advancing them in single file to secondary storage means provided with a [removable] *movable* gate normally acting to prevent said articles from escaping therefrom, a photoelectric tube, a light source providing a beam of light directed at said photoelectric tube across the path traveled by said removed articles as they advance in single file to said secondary storage means, electronic counting means coupled to said photoelectric tube and adapted to provide an output signal when said light beam has been interrupted a predetermined number of times by successive ones of said removed articles advancing in single file to said secondary storage means, means responsive to said output signal for moving said gate to release articles accumulated within said secondary storage means and for reclosing said gate thereafter to allow additional articles to accumulate, a packaging machine positioned to receive articles discharged from said secondary storage means, said packaging machine comprising means for forming a sealed package about said articles, drive means for said package forming means, electrically operated means responsive to said output signal for actuating said drive means in timed relation with movement of said gate, and means for terminating operation of said drive means after said articles have been packaged.

2. Apparatus for counting articles and packaging said articles in predetermined quantities as defined in claim 1, further including means connected in the output circuit of said counting means for varying the time duration of each signal passing from said counting means to said electrically operated means.

3. Apparatus for counting articles and segregating said articles in groups of identical predetermined number, said apparatus comprising a storage hopper for said articles having an opening through which articles are discharged therefrom, an endless belt conveyor positioned to receive articles discharged from said hopper, said conveyor having an upper article-conveying run traveling in a horizontal plane from an article-receiving position adjacent to said hopper to a predetermined article-discharging position and a lower run traveling from said article-discharging position back to said article-receiving position, a pair of spaced parallel guides located above said upper run, said guides having vertical flanges on their adjacent edges defining a channel between which articles on said upper run advance in single file to said article-discharging position, a vertically-extending housing open at its top and bottom ends, means supporting said housing with its open top end in position to receive articles falling free under the influence of gravity after discharge from said conveyor, a shutter mounted in said housing adjacent its bottom end, said shutter comprising a horizontal shaft extending transversely of said housing and diametrically-opposed radially extending planar members affixed to said shaft, a pair of bearings secured to said housing rotatably supporting the opposite ends of said shaft, a first rotary-acting solenoid connected to one end of said shaft for rotating said shaft in a first direction to shift said planar members from a first inclined position to a second inclined positon, a second rotary-acting solenoid identical to said first solenoid attached to the opposite end of said shaft for rotating said shaft in a second opposite direction to shift said planar members back to said first inclined position, said planar members having a length running radially from the axis of said shaft of sufficient magnitude to divert falling articles to one side of said housing when in said first inclined position and to the other side of said housing when in said second inclined position, a photoelectric tube, a light source providing a beam of light directed at said photo-electric tube across the free-fall path traveled by said articles between said conveyor and said shutter, electronic counting means coupled to said photoelectric tube for providing an output signal when said light beam has been interrupted a predetermined number of times by said free-falling articles, and means for energizing said first and second solenoids alternately in response to successive output signals.

4. Apparatus as defined by claim 3, further including first and second inclined inwardly extending plates attached to opposite walls of said housing at its bottom end, said first and second plates having their inner edges spaced to define a restricted article-discharge opening, said first plate having its inner edge positioned to be engaged by one of said planar members of said shutter when said planar members are in said first inclined position and said

second plate having its inner edge positioned to be engaged by said one planar member when said planar members are in said second inclined position.

5. Apparatus for counting articles and delivering them to a desired location in groups of identical predetermined number comprising, means for storing a supply of said articles, a feed roll for removing said articles from said storage means, said feed roll tapering inwardly from its ends to an intermediate constant diameter section provided with stepped surfaces for picking up articles in said storage means, means for separating said removed articles, means for advancing said separated articles in single file along a horizontal path to a point of discharge where said articles fall freely under the influence of gravity, gate means normally positioned in the path of said free-falling articles, whereby to cause said articles to accumulate in a group upstream of said gate, means for detecting each article as it falls freely under the influence of gravity, and means for opening said gate when a predetermined number of articles have been detected whereby to release the articles that have accumulated upstream of said gate, and for reclosing said gate to allow additional articles to accumulate.

6. Apparatus for counting articles and segregating them in groups, comprising means for separating a plurality of articles and advancing them in single file along a horizontal path to a point of discharge where they fall freely under the influence of gravity, an article-diverting shutter including a horizontally extending shaft positioned in the path of said free-falling articles, bearing rotatably supporting said shaft, first and second rotary acting solenoids connected to opposite ends of said shaft, said first solenoid when energized acting to rotate said shaft to turn said shutter from a first article-diverting position to a second article-diverting position, said second solenoid when energized acting to rotate said shaft to return said shutter to said first position, means for detecting each article as it falls freely under the influence of gravity, and means for energizing said first solenoid when a predetermined number of articles have been detected with said shutter in said first position and for energizing said second solenoid when a predetermined number of articles have been detected with said shutter in said second position.

7. Apparatus for counting articles and delivering them to a desired location in groups of identical predetermined number comprising, means for storing a supply of articles to be counted, means for transporting articles from said storage means along a horizontal path to a point of discharge where said articles fall freely under the influence of gravity, said article transporting means comprising an endless belt conveyor having a horizontal upper run, means for discharging articles from said storage means onto said upper run of said belt conveyor, first and second guide means for channeling articles into single file on said upper run, said first and second guide means comprising first portions running from said point of discharge toward said storage means and second portions continuous with said first portions at the end nearest said storage means, said first portions residing in a horizontal plane above said upper run and having parallel spaced vertically depending flanges extending longitudinally of said conveyor, said flanges defining a channel therebetween in which articles carried by said upper run can move only in single file as they advance toward said point of discharge, said flanges having bottom edges terminating just short of contact with said upper run, said second portions being set at a compound angle to said first portions, sloping downwardly toward the center line of said upper run and sloping also toward said storage means, said second portions having edges continuous with said bottom edge of said depending flanges, whereby articles discharged from said storage means onto said upper run are deflected by said second portions toward the center of said upper run for single-file movement between said depending flanges toward said point of discharge, a shutter positioned in the path of said free-falling article-diverting position, second means for turning said shutter to a second article-diverting position, means for detecting each article as it falls freely under the influence of gravity, means for producing an output signal each time a predetermined number of articles have been detected, and means for actuating said first and second shutter turning means alternately in response to successive output signals.

8. Apparatus for counting articles and segregating them into groups each totalling a predetermined number, said apparatus comprising means for storing a supply of articles to be counted, means for removing articles from said storage means and for advancing them in single file along a substantially horizontal path to a point of discharge where they fall freely under the influence of gravity, gate means normally positioned in the path of said free-falling articles, said gate comprising a horizontal shaft, bearings rotatably supporting said shaft, a pair of oppositely disposed blades secured to said shaft, a first solenoid connected to one end of said shaft for rotating said shaft to turn said blades back to said first inclined position, means for detecting each article as it falls freely from said point of discharge to said gate, means for generating an electric signal when a predetermined number of articles have been detected, and means for energizing said solenoids sequentially in response to successive output signals.

9. Apparatus as defined by claim 8 further including stationary means cooperating with said gate to cause free-falling articles to accumulate upstream of said gate only when said blades are in said first inclined position.

10. Apparatus as defined by claim 9 further including second stationary means cooperating with said gate to cause free-falling articles to accumulate upstream of said gate only when said blades are in said second inclined position.

11. In apparatus for counting and batching free-falling articles, the improvement comprising a stationary housing having a plurality of wall members defining an open top through which said articles may enter and an open bottom through which said articles may leave, at least two of said walls being in opposed parallel spaced relation to each other, a horizontal shaft extending through said two walls, bearing means rotatably supporting said shaft, a pair of opposed radially extending blades secured to said shaft within said housing, a first rotary-acting solenoid operatively connected to one end of said shaft for rotating said shaft to turn said blades to a first inclined position wherein articles are diverted laterally in a first direction, and a second rotary-acting solenoid operatively connected to the opposite end of said shaft for rotating said shaft to turn said blades to a second inclined position wherein articles are diverted laterally in a second direction.

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