

**(12) DEMANDE DE BREVET CANADIEN
CANADIAN PATENT APPLICATION**

(13) A1

(86) Date de dépôt PCT/PCT Filing Date: 2007/03/16
(87) Date publication PCT/PCT Publication Date: 2007/09/27
(85) Entrée phase nationale/National Entry: 2008/07/29
(86) N° demande PCT/PCT Application No.: EP 2007/052526
(87) N° publication PCT/PCT Publication No.: 2007/107521
(30) Priorité/Priority: 2006/03/22 (DE10 2006 013 069.3)

(51) Cl.Int./Int.Cl. *B64C 1/00* (2006.01),
B64C 5/06 (2006.01), *F16B 2/04* (2006.01),
F16B 43/02 (2006.01)

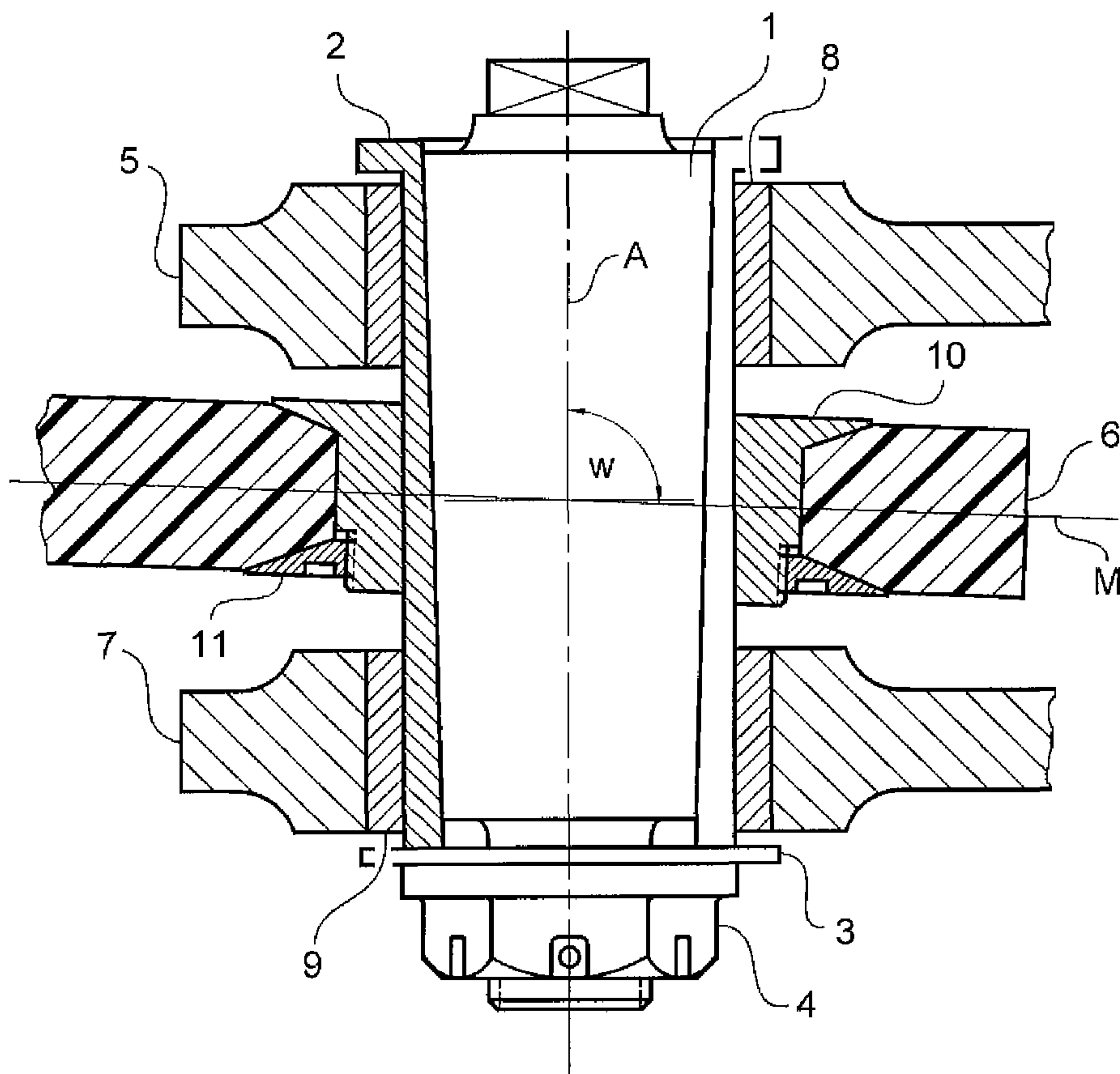
(71) Demandeur/Applicant:
AIRBUS DEUTSCHLAND GMBH, DE

(72) Inventeur/Inventor:
STEINKE, UWE, DE

(74) Agent: OYEN WIGGS GREEN & MUTALA LLP

(54) Titre : JOINT BOULONNE

(54) Title: BOLTED JOINT



(57) Abrégé/Abstract:

(57) *Abgeger, Abstract.*
In double-shear bolted joints for connecting load-transferring structural parts on an aircraft, wherein a first structural part having a bolt eye with a bushing is connected by means of a bolt to a second, generally fork-shaped structural part having two bolt eyes and

(57) Abrégé(suite)/Abstract(continued):

the first structural part acts upon the bolt at an angle $\neq 90^\circ$, it was previously disadvantageous that, in the surrounds of the particular bolt eye with the bushing, a material thickening was necessary. In order to avoid this thickening, it is proposed that the first structural part (6) has a constant thickness and the bushing (10) is inserted perpendicular to the surface of the structural part (6), the axis (A) of the bore within the bushing (10) for receiving the bolt (1) running obliquely to the surface of the structural part (6) and obliquely to the end face of the bushing (10). The result is that advantages are obtained in terms of manufacturing costs and weight of the bolted joint.

(12) INTERNATIONAL APPLICATION PUBLISHED UNDER THE PATENT COOPERATION TREATY (PCT)

(19) World Intellectual Property Organization
International Bureau(43) International Publication Date
27 September 2007 (27.09.2007)

PCT

(10) International Publication Number
WO 2007/107521 A1

(51) International Patent Classification:

B64C 1/00 (2006.01) **F16B 2/04** (2006.01)
B64C 5/06 (2006.01) **F16B 43/02** (2006.01)

(81) **Designated States** (unless otherwise indicated, for every kind of national protection available): AE, AG, AL, AM, AT, AU, AZ, BA, BB, BG, BR, BW, BY, BZ, CA, CH, CN, CO, CR, CU, CZ, DE, DK, DM, DZ, EC, EE, EG, ES, FI, GB, GD, GE, GH, GM, GT, HN, HR, HU, ID, IL, IN, IS, JP, KE, KG, KM, KN, KP, KR, KZ, LA, LC, LK, LR, LS, LT, LU, LY, MA, MD, MG, MK, MN, MW, MX, MY, MZ, NA, NG, NI, NO, NZ, OM, PG, PH, PL, PT, RO, RS, RU, SC, SD, SE, SG, SK, SL, SM, SV, SY, TJ, TM, TN, TR, TT, TZ, UA, UG, US, UZ, VC, VN, ZA, ZM, ZW.

(21) International Application Number:

PCT/EP2007/052526

(22) International Filing Date: 16 March 2007 (16.03.2007)

(25) Filing Language: English

(26) Publication Language: English

(30) Priority Data:
10 2006 013 069.3 22 March 2006 (22.03.2006) DE(71) Applicant (for all designated States except US): **AIRBUS DEUTSCHLAND GMBH** [DE/DE]; Kreetslag 10, 21129 Hamburg (DE).

(84) **Designated States** (unless otherwise indicated, for every kind of regional protection available): ARIPO (BW, GH, GM, KE, LS, MW, MZ, NA, SD, SL, SZ, TZ, UG, ZM, ZW), Eurasian (AM, AZ, BY, KG, KZ, MD, RU, TJ, TM), European (AT, BE, BG, CH, CY, CZ, DE, DK, EE, ES, FI, FR, GB, GR, HU, IE, IS, IT, LT, LU, LV, MC, MT, NL, PL, PT, RO, SE, SI, SK, TR), OAPI (BF, BJ, CF, CG, CI, CM, GA, GN, GQ, GW, ML, MR, NE, SN, TD, TG).

(72) Inventor; and

Published:

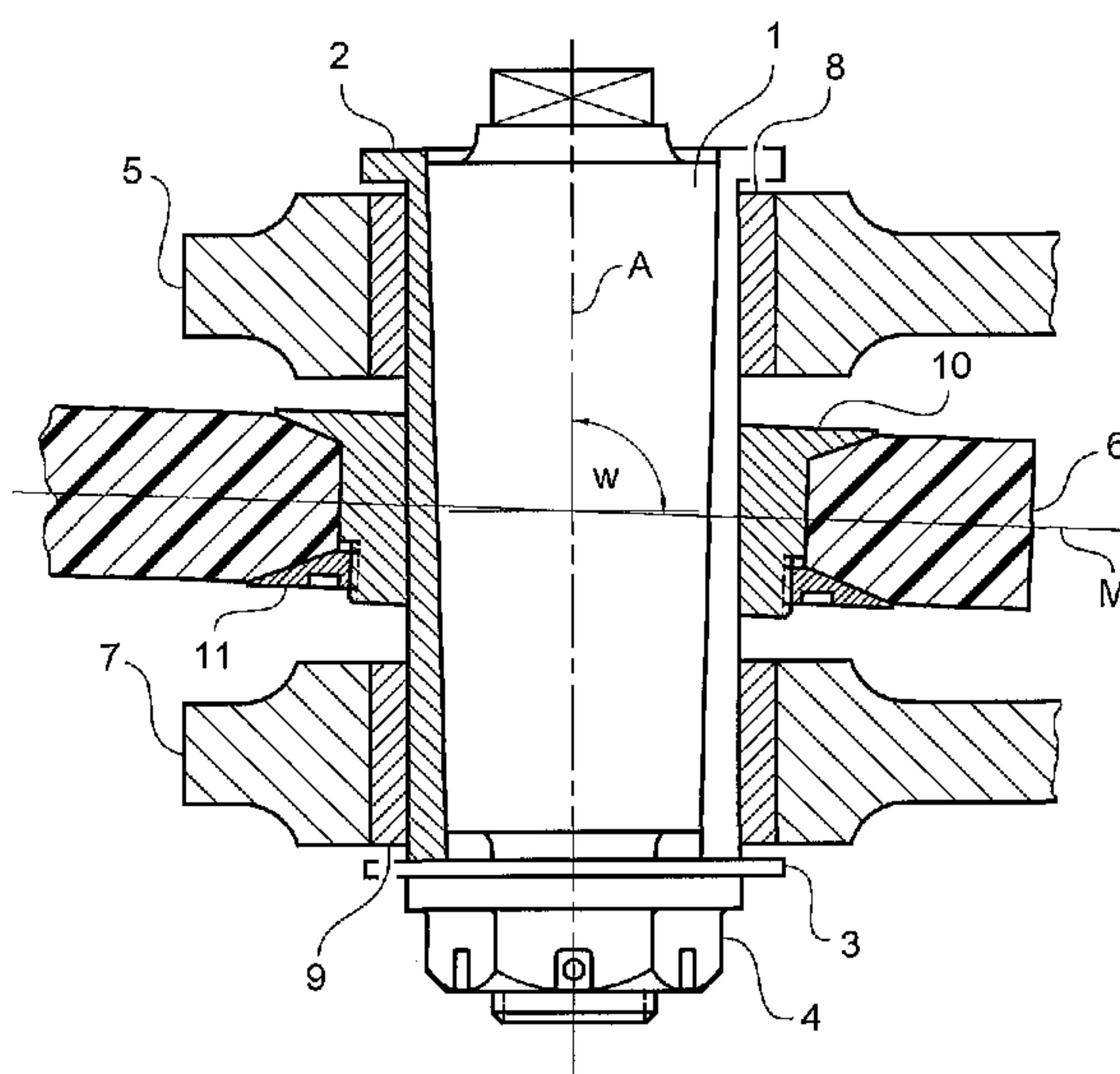
(75) Inventor/Applicant (for US only): **STEINKE, Uwe** [DE/DE]; Heidblick 21, 21149 Hamburg (DE).

— with international search report

(74) Agent: **PECKMANN, Ralf**; Reinhard, Skuhra, Weise & Partner GbR, Friedrichstrasse 31, 80801 München (DE).

For two-letter codes and other abbreviations, refer to the "Guidance Notes on Codes and Abbreviations" appearing at the beginning of each regular issue of the PCT Gazette.

(54) Title: BOLTED JOINT



(57) **Abstract:** In double-shear bolted joints for connecting load-transferring structural parts on an aircraft, wherein a first structural part having a bolt eye with a bushing is connected by means of a bolt to a second, generally fork-shaped structural part having two bolt eyes and the first structural part acts upon the bolt at an angle $\neq 90^\circ$, it was previously disadvantageous that, in the surrounds of the particular bolt eye with the bushing, a material thickening was necessary. In order to avoid this thickening, it is proposed that the first structural part (6) has a constant thickness and the bushing (10) is inserted perpendicular to the surface of the structural part (6), the axis (A) of the bore within the bushing (10) for receiving the bolt (1) running obliquely to the surface of the structural part (6) and obliquely to the end face of the bushing (10). The result is that advantages are obtained in terms of manufacturing costs and weight of the bolted joint.

WO 2007/107521 A1

Bolted joint

The invention relates to a bolted joint for connecting load-transferring structural parts on an aircraft. Such joints are generally of double-shear configuration, a first structural part having a bolt eye being connected by means of a bolt to a second, generally fork-shaped structural part having two bolt eyes. In general, the direction of the loads transferred through the bolt from the structural parts runs at right angles to the bolt axis. Occasionally, for design reasons, bolted joints can also be realized in which the direction of the load arriving through the first structural part forms an angle $\neq 90^\circ$ with the bolt axis. This means that on the side of the first structural part the bore for receiving the bolt should be made at an angle $\neq 90^\circ$, i.e. obliquely to the structural part or obliquely to the surface thereof. In order to achieve this, the surrounds of the bore are previously thickened and this local thickening relative to the rest of the structural part is provided with a bevel precisely corresponding to the necessary angle. As a result, the boring for making the receiving bore for the bolt can always be carried out locally perpendicular to the structural part surface, which is very desirable for methodological reasons. The said thickening is necessary because other measures, e.g. milling, for creating the local bevel would lead to a weakening of the structural part. Depending on the material of the structural parts, different bushing versions are used. If the structural parts consist of metal, then simple bushings of suitable material are forced in. If the structural parts consist of a fibre composite, however, then it is expedient to use so-called screw bushings. Owing to their shape, these allow the structural part material to be well supported in the axial direction.

In the case of structural parts corresponding to the aforementioned thickening, it is disadvantageous that the creation of the thickening involves a considerable amount of work, especially when the structural parts consist of fibre composite. In addition, it is disadvantageous that the thickening entails a longer bushing and hence also a longer bolt than in a struc-

tural part with no thickening. The longer bolt and the longer bushing give rise, in the first place, to a higher weight of the bolted joint. In the second place, in the case of a longer bolt, larger distances between the particular load-application 5 points are obtained, whereby higher bending moments act upon the bolt than in a structural part with no thickening.

The object of the invention is therefore to refine a bolted joint according to the stated prior art in such a way that a 10 thickening of the particular structural part, given an oblique position of the bolt axis relative to the structural part, is avoided.

This object is achieved according to Claim 1 by the fact that 15 the structural part has a constant thickness and the bushing is inserted perpendicular to the surface of the structural part, the axis of the bore within the bushing for receiving the bolt running obliquely to the surface of the structural part and obliquely to the end face of the bushing.

20

Advantageous embodiments of the invention are defined in the sub-claims.

By virtue of the invention:

25 - the production-engineering input,
- the length of the bushing,
- the length of the bolt,
- the weight of the bolted joint, and
- the bending load upon the bolt
30 are reduced. This yields advantages in terms of the costs and weight of the bolted joint.

The invention is represented in the drawing and explained in greater detail with reference to the description.

35

Fig. 1 shows a bolted joint in a sectional representation, comprising a bolt 1, a slotted sleeve 2, a washer 3 and a nut 4. The bolt 1 with the sleeve 2 respectively passes through a structural part 5, a structural part 6 and a structural part 7 .

and is secured by the washer 3 and the nut 4. The structural parts 5 and 7 are here elements of a fork, not shown, and consist of a metallic material. The bores present in the structural parts 5, 6 and 7 for receiving the sleeve 2 with the bolt 1 respectively form a bolt eye reinforced by a metal bushing 8, 9 and 10. The bushings 8 and 9 consist of suitable material, have a cylindrical shape and are pressed into the particular structural parts 5 and 7. In the illustrated embodiment, the structural part 6 consists of a fibre composite, for which reason there is here provided a screw bushing 10 with a nut 11. The screw bushing 10 is inserted perpendicular to the surface of the structural part 6 of constant thickness. The bore within the bushing 10 for receiving the bolt 1 runs obliquely to the surface of the structural part 6 and at the same angle obliquely to the end face of the bushing 10. The axis A shown in the diagram is thus simultaneously the axis of the bolt 1 and of the said bore. As a result of the shape of the screw bushing 10 with the nut 11, good support is given to the structural part material in the axial direction. The bolt 1 has a conical shape, against which the inner surface of the sleeve 2 comes to bear. For the assembly of the bolted joint, the sleeve 2 is firstly put through the bolt eyes of the structural parts 5, 6 and 7. The bolt 1 is then introduced into the sleeve 2 and secured by means of the washer 3 and the nut 4. Owing to the conical shape of the bolt 1 and the slotted configuration of the sleeve 2, the bolt 1 and the sleeve 2 interact in such a way that the outer diameter of the sleeve 2 enlarges and the outer surface thereof comes firmly to bear against the inner surface of the bushings 8, 9 and 10 when the nut 4 is tightened. The illustrated section is run through the slot of the sleeve 2, so that only that region of the sleeve which can be seen on the left in the diagram appears in hatched representation. In the assembly operation, the nut 4 is tightened with a predetermined torque, whereby a defined radial force is exerted upon the bushings 8, 9 and 10 and thus a non-positive connection is formed for the transfer of loads. The centre plane of the structural part 6 is represented by a line denoted by M. This intersects the axis A of the bolt 1 at an angle W, which, owing to the oblique position of the struc-

tural part 6, which is necessary for design reasons, has a defined value $\neq 90^\circ$.

One embodiment of the invention consists in the bolt 1 being 5 cylindrically configured and directly touching the bushings 8, 9 and 10. As a result of the inventive oblique arrangement of the bore for receiving the bolt in the bushing 10, the previous thickening in the region of the particular bolt eye is dispensed with. The aforementioned advantageous effects are 10 thereby achieved, with positive repercussions upon manufacturing costs and weight of the bolted joint.

Patent Claims

1. Bolted joint for connecting load-transferring structural parts on an aircraft, which bolted joint is of double-shear configuration, a first structural part having a bolt eye being connected by means of a bolt to a second, generally fork-shaped structural part having two bolt eyes, characterized in that the first structural part (6) has a constant thickness and the bushing (10) is inserted perpendicular to the surface of the structural part (6), the axis (A) of the bore within the bushing (10) for receiving the bolt (1) running obliquely to the surface of the structural part (6) and obliquely to the end face of the bushing (10).
- 15 2. Bolted joint according to Claim 1, characterized in that the bushing (10) is configured as a screw bushing with a nut (11).
- 20 3. Bolted joint according to Claim 1 or 2, characterized in that the bolt (1) is conically configured and interacts with an internally conical, slotted sleeve (2).

Fig. 1

