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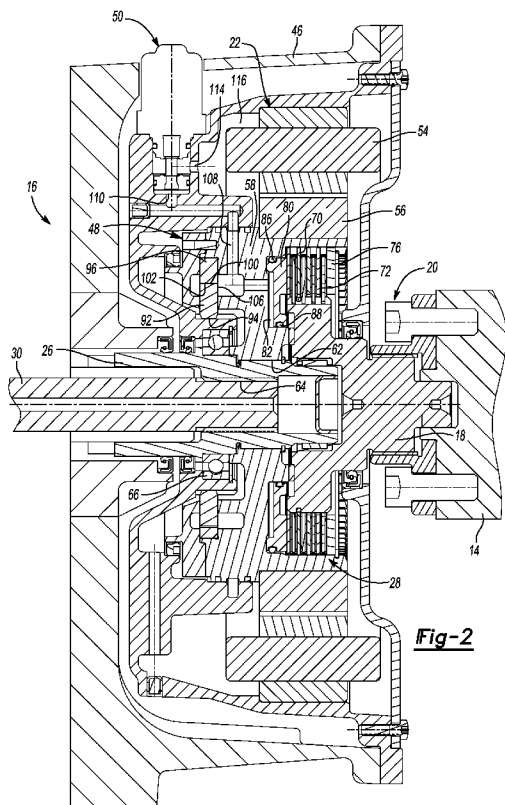
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(54) Title: CONTROLLED GEROTOR ACTUATED PRE-TRANS PARALLEL HYBRID



(57) Abstract: A pre-transmission unit for a hybrid drive system includes an input shaft adapted to be driven by an internal combustion engine. An electric motor includes a rotor fixed for rotation with an output shaft, and a stator fixed to a housing. A multi-plate clutch is positioned in the housing to drivingly interconnect the input shaft and the rotor. The clutch includes a piston for applying an input force to the clutch plates. A pump is positioned in the housing and includes an input member fixed for rotation with the rotor. The pump provides pressurized fluid to the piston. A valve regulates the fluid pressure applied to the piston and varies the torque transferred by the clutch.



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CONTROLLED GEROTOR ACTUATED PRE-TRANS PARALLEL HYBRID
CROSS-REFERENCE TO RELATED APPLICATIONS

[0001] This application claims the benefit of U.S. Provisional Application No. 61/426,816, filed on December 23, 2010. The entire disclosure
5 of the above application is incorporated herein by reference.

FIELD

[0002] The present disclosure generally relates to a hybrid vehicle drivetrain. More particularly, a pre-transmission unit is positioned downstream from an internal combustion engine and upstream from a power transfer unit.
10 The pre-transmission unit includes an electric motor, a clutch and a fluid pump for controlling the clutch.

BACKGROUND

[0003] Vehicle manufacturers have been diligently working to develop alternative powertrain systems in an effort to reduce the level of pollutants
15 exhausted into the air by conventional powertrains equipped with internal combustion engines and also to increase the fuel efficiency thereof. Significant development has been directed to hybrid vehicles. Several different hybrid electric vehicles are configured with an internal combustion and an electric motor that may be operated independently or in combination to drive the vehicle.

[0004] In general, there are two types of hybrid vehicles, namely, series hybrid and parallel hybrid. In a series hybrid vehicle, power is delivered to the wheels by the electric motor which draws electrical energy from a battery. The engine is used in a series hybrid vehicle to drive a generator which supplies power directly to the motor or charges the battery when a state of charge falls
20 below a predetermined value. In a parallel hybrid vehicle, the electric motor and the internal combustion engine may be operated independently or in combination with one another.

[0005] In one known hybrid vehicle powertrain, the valve train of the internal combustion engine is controlled such that air is not compressed during a
30 compression stroke of the internal combustion engine when power is provided solely from the electric motor. While this configuration may function properly,

such an arrangement may be very costly and complex. As such, it may be desirable to provide a simplified hybrid drive system.

SUMMARY

5 **[0006]** This section provides a general summary of the disclosure, and is not a comprehensive disclosure of its full scope or all of its features.

10 **[0007]** A pre-transmission unit for a hybrid drive system includes an input shaft adapted to be driven by an internal combustion engine. An electric motor includes a rotor fixed for rotation with an output shaft, and a stator fixed to a housing. A multi-plate clutch is positioned in the housing to drivingly interconnect the input shaft and the rotor. The clutch includes a piston for applying an input force to the clutch plates. A pump is positioned in the housing and includes an input member fixed for rotation with the rotor. The pump provides pressurized fluid to the piston. A valve regulates the fluid pressure applied to the piston and varies the torque transferred by the clutch.

15 **[0008]** A method of operating a pre-transmission unit in a hybrid drive system of a vehicle includes positioning the pre-transmission unit downstream of an internal combustion engine and upstream from a transmission. The method includes energizing an electric motor of the pre-transmission unit to provide torque to the transmission and drive a pump positioned within the pre-
20 transmission unit. A valve in receipt of pressurized fluid from the pump is controlled to regulate a fluid pressure provided to a clutch that drivingly interconnects the internal combustion engine with the electric motor and the transmission. The valve is opened to cease torque transmission across the clutch to disconnect the internal combustion engine from the electric motor and
25 the transmission.

30 **[0009]** Further areas of applicability will become apparent from the description provided herein. The description and specific examples in this summary are intended for purposes of illustration only and are not intended to limit the scope of the present disclosure.

DRAWINGS

[0010] The drawings described herein are for illustrative purposes only of selected embodiments and not all possible implementations, and are not intended to limit the scope of the present disclosure.

5 **[0011]** Figure 1 is a schematic depicting a vehicle equipped with the pre-transmission unit of the present disclosure;

[0012] Figure 2 is a cross-sectional side view of the pre-transmission unit; and

10 **[0013]** Figure 3 is a fragmentary perspective view of the pre-transmission unit.

[0014] Corresponding reference numerals indicate corresponding parts throughout the several views of the drawings.

DETAILED DESCRIPTION

15 **[0015]** Example embodiments will now be described more fully with reference to the accompanying drawings.

[0016] The present disclosure relates to a hybrid drive system for a vehicle which functions to provide driving torque to one or more ground-engaging wheels of a vehicle. The hybrid drive system described relates to a parallel hybrid electric vehicle employing an electric motor and an internal combustion engine. The internal combustion engine may be a gasoline engine, a diesel engine, turbine engine, or the like. The electric motor is arranged as a portion of a pre-transmission unit located downstream of the internal combustion engine and upstream of a power transfer unit. The pre-transmission unit also includes an engine disconnect clutch and a pump.

25 **[0017]** With reference to Figure 1, a hybrid drive system 10 includes an internal combustion engine 12 providing output torque to a crankshaft 14. A pre-transmission unit 16 includes an input shaft 18 fixed for rotation with crankshaft 14 via a coupling 20. Pre-transmission unit 16 includes an electric motor 22 operable to transmit torque to an output shaft 26 of pre-transmission unit 16.
30 Pre-transmission unit 16 also includes a clutch 28 for drivingly interconnecting crankshaft 14 to pre-transmission unit output shaft 26. An input shaft 30 of a power transfer unit 32 is drivingly coupled to output shaft 26. Power transfer unit

32 may be constructed as a transmission of the manual or automatic type to provide driving torque to an axle assembly 36. Axle assembly 36 may include a differential assembly 38 for distributing torque to wheels 40. It should be appreciated that power transfer unit 32 may also be structured as a transfer case to selectively transfer torque to drive axle 36 alone or also transfer torque to another drive axle not shown in the Figure.

[0018] With reference to Figures 2 and 3, pre-transmission unit 16 includes a housing 46 containing electric motor 22, clutch 28, a gerotor pump 48 and a valve 50. Electric motor 22 includes a stator 54 fixed to housing 46 and a rotor 56. Rotor 56 is fixed for rotation with a hub 58. Hub 58 is rotatable relative to housing 46. Hub 58 is fixed for rotation with output shaft 26 via a spline 62. Output shaft 26 is fixed for rotation with input shaft 30 via a spline 64. A bearing 66 rotatably supports output shaft 26 and hub 58 within housing 46.

[0019] Clutch 28 includes a plurality of outer clutch plates 70 interleaved with a plurality of inner clutch plates 72. Outer clutch plates 70 are axially moveable relative to and fixed for rotation with hub 58. A reaction plate 76 is also fixed for rotation with hub 58. Inner clutch plates 72 are axially moveable relative to and fixed for rotation with input shaft 18. An apply plate 80 is positioned within a cavity 82 formed in hub 58. First and second seals 86, 88 are coupled to apply plate 80 allowing the apply plate to function as a piston. As will be described in greater detail hereinafter, pressurized fluid may be selectively provided to cavity 82 to cause apply plate 80 to transfer an input force to the interleaved plates of clutch 28.

[0020] Gerotor pump 48 includes an inner pumping member 92 fixed to housing 46 at a splined connection 94 as well as an outer pumping member 96. Outer pumping member 96 includes a plurality of lobes 100 cooperating with a plurality of lobes 102 formed on inner pumping member 92. Relative rotation between inner pumping member 92 and outer pumping member 96 causes pressurized fluid to exit pump 48 at an outlet port 106. Outlet port 106 is in fluid communication with cavity 82. Outlet port 106 is also in fluid communication with a passageway 108 extending through hub 58 and housing 46 to a valve inlet

110. An outlet 114 of valve 50 is in fluid communication with a sump 116 within housing 46.

[0021] Valve 50 is operable to open and close an internal passageway interconnecting inlet 110 and outlet 114. A controller 120 is in communication with engine 12 and pre-transmission unit 16. A battery 124 provides energy to controller 120 and electric motor 22. Controller 120 is configured to provide a pulse width modulation signal to valve 50 to control the pressure provided to cavity 82 acting on apply plate 80. In this manner, the torque transferred by clutch 28 may also be varied. Valve 50 may be operated in a completely open mode as well as a completely closed mode in addition to using pulse width modulation control.

[0022] Hybrid drive system 10 is operable in several modes where either electric motor 22, internal combustion engine 12 or the combination of both internal combustion engine 12 and electric motor 22 provide drive torque to power transfer unit 32. Clutch 28 is provided to disconnect crankshaft 14 of internal combustion engine 12 from rotor 56 of electric motor 22. When hybrid drive system 10 operates in a purely electric drive mode, crankshaft 14, and the associated internal components of internal combustion engine 12 are not moved. Pumping losses associated with rotating the internal combustion engine are eliminated.

[0023] Clutch 28 may also serve as a launch clutch when it is desired to start the vehicle from a stopped position using either or both internal combustion engine 12 and electric motor 22. Pump 48 provides pressurized fluid to clutch 28 any time relative motion occurs between hub 58 and housing 46. As previously mentioned, transmission input shaft 30 is also fixed for rotation with hub 58. Accordingly, as transmission input shaft 30 or rotor 56 rotates, pump 48 outputs pressurized fluid to cavity 82 engaging apply plate 80 with inner and outer plates 72, 70 of clutch 28. Controller 120 controls valve 50 to regulate the pressure provided to apply plate 80 and vary the torque transferred across clutch 28.

[0024] To improve energy efficiency of hybrid drive system 10, internal combustion engine 12 may be frequently stopped and started. Electric motor 22

and clutch 28 may be used as a starter motor to rotate crankshaft 14 and start internal combustion engine 12. Alternatively, if the vehicle equipped with hybrid drive system 10 is moving, clutch 28 may be used to transfer driveline torque from the vehicle wheels through power transfer unit 32 and clutch 28 to crankshaft 14. Controller 120 and valve 50 function to smoothly transfer energy to crankshaft 14 to allow an internal combustion engine starting procedure with minimal noise, vibration or harshness transferred to the vehicle occupants.

[0025] Other modes of operating hybrid drive system 10 include operating electric motor 22 as a generator in a regenerative braking mode of operation or simply a battery charging mode of operation where internal combustion engine 12 drives not only wheels 40 but also rotor 56 to charge battery 124.

[0026] By configuring pre-transmission unit 16 in the manner described, the total number of components used to provide the various functions is reduced when compared to the prior art. The overall package size required is also minimized. The compact packaging is at least partially accomplished by sizing rotor 56 to encompass clutch 28 and positioning gerotor pump 48 immediately adjacent electric motor 22 and clutch 28. To further condense the size of pre-transmission unit 16, each of the rotating components of electric motor 22, clutch 28 and pump 48 rotate about a common axis of rotation as crankshaft 14 and transmission input shaft 30.

[0027] The foregoing description of the embodiments has been provided for purposes of illustration and description. It is not intended to be exhaustive or to limit the disclosure. Individual elements or features of a particular embodiment are generally not limited to that particular embodiment, but, where applicable, are interchangeable and can be used in a selected embodiment, even if not specifically shown or described. The same may also be varied in many ways. Such variations are not to be regarded as a departure from the disclosure, and all such modifications are intended to be included within the scope of the disclosure.

CLAIMS

What is claimed is:

- 5 1. A pre-transmission unit for a hybrid drive system, comprising:
a housing;
an input shaft adapted to be driven by an internal combustion engine;
an output shaft;
an electric motor including a rotor fixed for rotation with the output shaft,
and a stator fixed to the housing;
- 10 a multi-plate clutch positioned in the housing and drivingly interconnecting
the input shaft and the rotor, the clutch including a piston for applying an input
force to the clutch plates;
a pump positioned in the housing and including an input member fixed for
rotation with the rotor, the pump providing pressurized fluid to the piston; and
- 15 a valve for regulating the fluid pressure applied to the piston and varying
the torque transferred by the clutch.
2. The pre-transmission unit of claim 1, wherein the pump input
member includes a multi-lobed gerotor rotor and the pump also including a multi-
20 lobed gerotor stator fixed to the housing.
3. The pre-transmission unit of claim 2, wherein the pump includes a
rotatable hub fixed to the gerotor rotor, the electric motor rotor and the output
shaft.
- 25 4. The pre-transmission unit of claim 3, wherein the hub includes a
fluid passageway interconnecting the piston and an outlet of the pump.
5. The pre-transmission unit of claim 4, wherein the valve is
30 positioned within the housing in fluid communication with a passageway
extending through the housing and the passageway through the hub to
interconnect the piston and the outlet of the pump.

6. The pre-transmission unit of claim 1, wherein the electric motor rotor circumscribes the clutch.

5 7. The pre-transmission unit of claim 1, wherein the clutch plates, the pump input member, the electric motor rotor, the input shaft, and the output shaft rotate about a common axis.

10 8. The pre-transmission unit of claim 1, wherein the valve is operable to open and close in response to a pulse width modulation signal and regulate the pressure of the fluid acting on the piston.

15 9. The pre-transmission unit of claim 8, wherein the valve allows pressurized fluid from the pump to enter a low pressure sump when open.

10. The pre-transmission unit of claim 1, wherein the electric motor is sized to launch a vehicle from a standstill.

20 11. The pre-transmission unit of claim 1, wherein the electric motor is operable as a generator.

12. A method of operating a pre-transmission unit in a hybrid drive system of a vehicle, the method comprising:

25 positioning the pre-transmission unit downstream of an internal combustion engine and upstream from a transmission;

energizing an electric motor of the pre-transmission unit to provide torque to the transmission and drive a pump positioned within the pre-transmission unit;

30 controlling a valve in receipt of pressurized fluid from the pump to regulate a fluid pressure provided to a clutch that drivingly interconnects the internal combustion engine with the electric motor and the transmission; and

opening the valve to cease torque transmission across the clutch to disconnect the internal combustion engine from the electric motor and the transmission.

5 13. The method of claim 12, further including transferring pressurized fluid through a rotatable hub to interconnect a piston of the clutch and an outlet of the pump.

10 14. The method of claim 13, further including circumscribing the clutch with the electric motor.

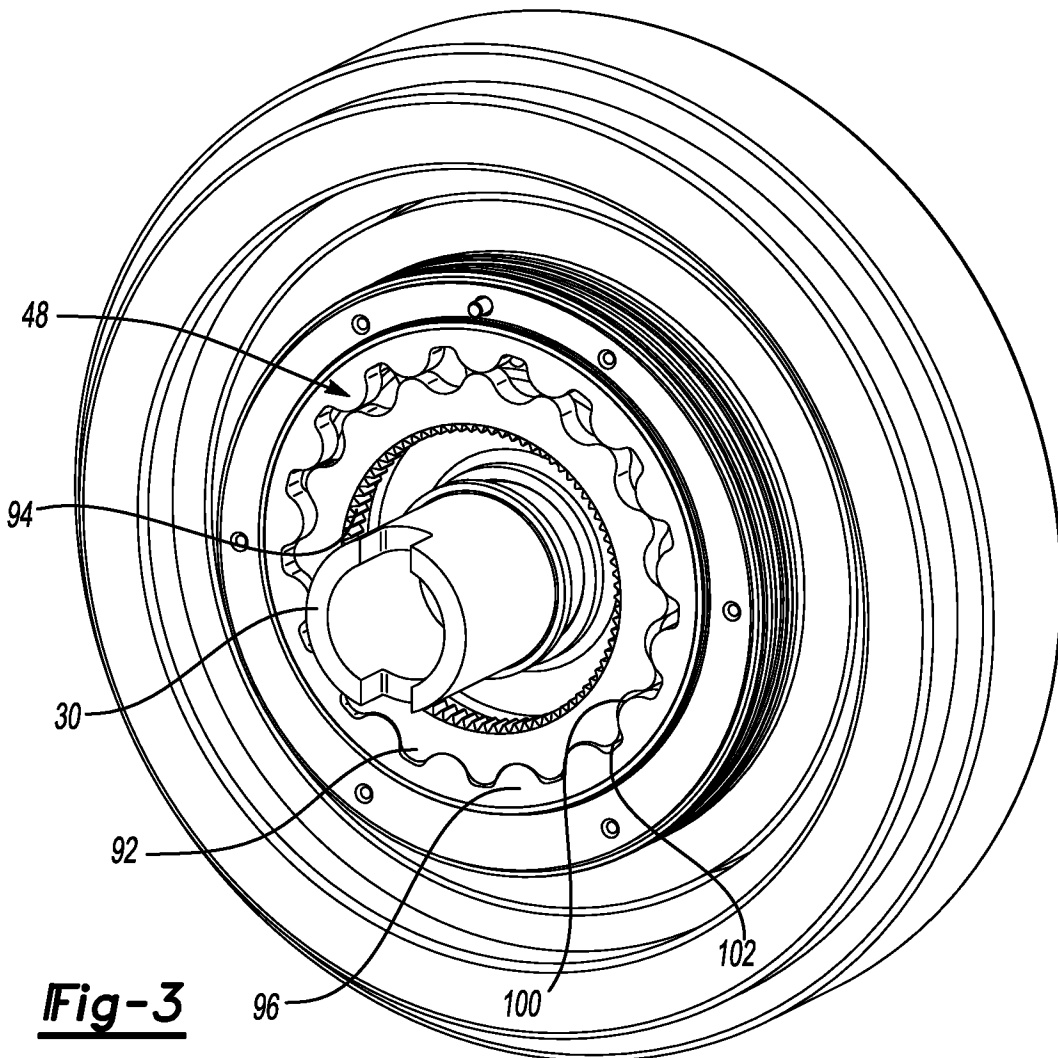
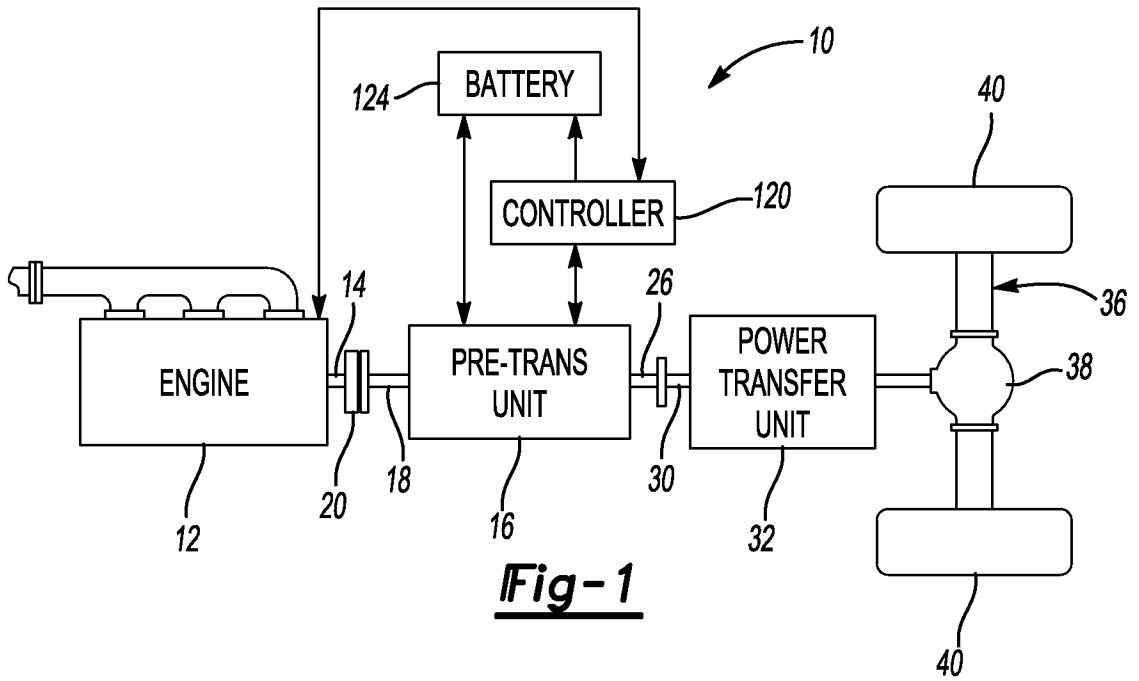
15 15. The method of claim 13, further including defining a portion of the pump with a recess formed in the hub.

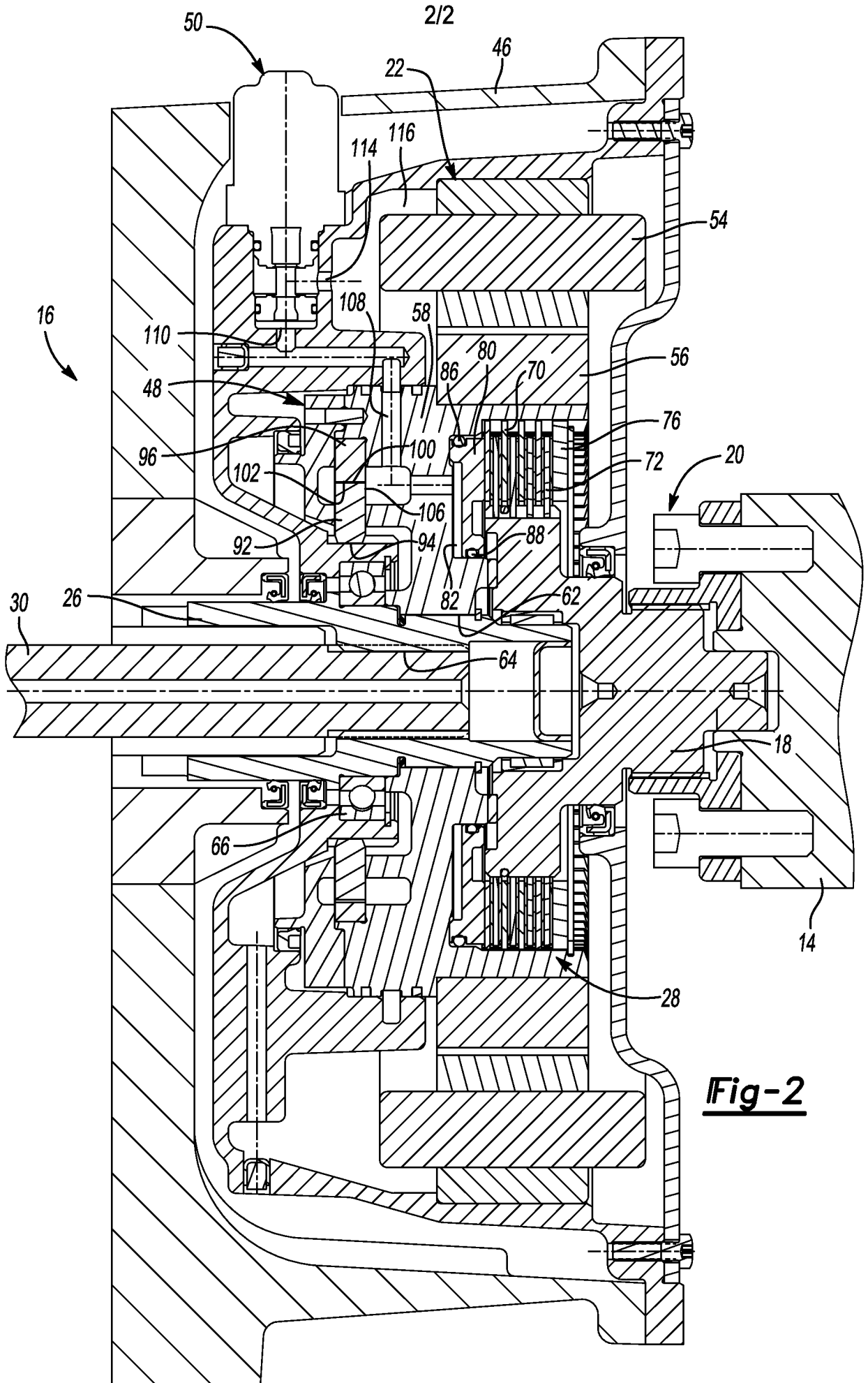
 16. The method of claim 12, further including driving the electric motor with the internal combustion engine to charge a battery.

20 17. The method of claim 12, further including driving the electric motor with kinetic energy from the vehicle to charge a battery.

 18. The method of claim 12, further including starting the internal combustion engine by energizing the electric motor and controlling the valve to increase a torque transferred across the clutch.

25 19. The method of claim 18, further including simultaneously transferring torque from the internal combustion engine and the electric motor to the transmission.





INTERNATIONAL SEARCH REPORT

International application No
PCT/US2011/064830

A. CLASSIFICATION OF SUBJECT MATTER					
INV.	B60K6/405	B60K6/387	B60K6/48	B60W30/18	B60W30/192
	B60W10/02	B60W20/00	B60W10/06	B60W10/08	F16D48/08
ADD.	F04C2/10				
According to International Patent Classification (IPC) or to both national classification and IPC					

B. FIELDS SEARCHED
Minimum documentation searched (classification system followed by classification symbols) B60K B60W F16D F04C

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

EPO-Internal

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	JP 2008 126702 A (MAZDA MOTOR) 5 June 2008 (2008-06-05) abstract; figures 1-14 -----	1,6-19
X	DE 10 2007 062237 A1 (DAIMLER AG [DE]) 25 June 2009 (2009-06-25) paragraphs [0052] - [0054]; figures 5-8 -----	1,6-19
A	US 2007/149338 A1 (EBNER NORBERT [DE] ET AL) 28 June 2007 (2007-06-28) the whole document -----	1,12

Further documents are listed in the continuation of Box C.
 See patent family annex.

* Special categories of cited documents :

<p>"A" document defining the general state of the art which is not considered to be of particular relevance</p> <p>"E" earlier document but published on or after the international filing date</p> <p>"L" document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)</p> <p>"O" document referring to an oral disclosure, use, exhibition or other means</p> <p>"P" document published prior to the international filing date but later than the priority date claimed</p>	<p>"T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention</p> <p>"X" document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone</p> <p>"Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art.</p> <p>"&" document member of the same patent family</p>
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Date of the actual completion of the international search <p style="text-align: center; font-size: 1.2em;">28 February 2012</p>	Date of mailing of the international search report <p style="text-align: center; font-size: 1.2em;">06/03/2012</p>
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Name and mailing address of the ISA/ European Patent Office, P.B. 5818 Patentlaan 2 NL - 2280 HV Rijswijk Tel. (+31-70) 340-2040, Fax: (+31-70) 340-3016	Authorized officer <p style="text-align: center; font-size: 1.2em;">Vogt-Schilb, Gérard</p>
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INTERNATIONAL SEARCH REPORT

Information on patent family members

International application No

PCT/US2011/064830

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