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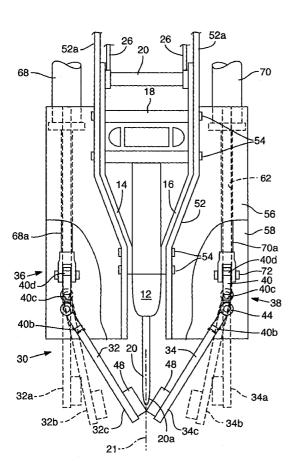
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[Continued on next page]

(54) Title: GRIPPING ASSEMBLY FOR IMPACT HAMMER



(57) Abstract: A gripping assembly mounted on an impact hammer (12) having a longitudinally extending impact axis, the gripping assembly including a pair of opposed elongate gripping arms (32, 34), having gripping end portions (42) which may be extended and swung toward each other to grip material in the region of the working end of the impact hammer. When retracted, the gripping arms rest to opposite sides of the impact hammer (12) to permit free operation of the impact hammer.

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GRIPPING ASSEMBLY FOR IMPACT HAMMER

CROSS-REFERENCE TO RELATED APPLICATION

This application claims priority to U.S. Provisional Application No. 60/284,388, filed April 16, 2001.

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FIELD OF THE INVENTION

This invention relates generally to material working equipment, more particularly to equipment in which a gripping assembly is adapted to be associated with an impact hammer to permit materials in the region of the impact hammer to be grasped and manipulated.

BACKGROUND

In mining, excavation, and other fields, it often is necessary to provide means for breaking and maneuvering, or manipulating, hardened materials, such as rock.

One example of an area in which such need exists is in mining applications in which excavated rock is dumped onto a large screening device known as a grizzly. Grizzlies often are recessed below working level to accommodate sidecar or truck dumping of material thereon. The grizzly may become clogged by dumping oversized rock or boulders thereon, by finer material bridging the openings in the grizzly, and by other types of debris that may have been mixed with the mined materials.

In the past, many mine operators have resorted to manual labor to clear material from the grizzlies. For example, workmen move into the grizzly region with sledgehammers, rope slings, etc. to break up oversized rock and to attempt to remove debris therefrom.

In other operations, permanently installed impact hammers are used to break the oversized rock, such that it may move through the pre-selected sized openings in the grizzly. However, they provide no means for grasping and manipulating stone or debris, other than to attempt to break it by use of the impact hammer.

In light of the number of accidents that have occurred in the use of manual labor to clear grizzlies, and the ineffectiveness of jackhammers alone, it has become important to provide some means for breaking hardened materials, such as rocks, and to manipulate or grasp materials in the region of the impact hammer.

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SUMMARY OF THE DISCLOSURE

In one aspect of the present disclosure, a gripping arm assembly is adapted to be mounted on an impact hammer having a longitudinally extending impact axis. The assembly includes a pair of opposed, laterally spaced, elongate gripping arms, each of which has a gripping portion adjacent one of its ends, mounting mechanism adapted to mount the arms on the impact hammer for movement between retracted, non-gripping positions, and extended gripping, or pinching, positions, and operator mechanism coupled to the gripping arms operable to shift them between their retracted and extended positions.

Another aspect of the disclosure is to provide a mounting mechanism which includes a track for each gripping arm for guiding movement of the gripping portion of the gripping arm as it is shifted between its retracted and extended positions.

Yet another aspect of the disclosure is to provide a gripping arm assembly adapted to be mounted on an impact hammer, in which a gripping arm includes a connector portion connected to operator mechanism and a swinging portion pivotally coupled to the connector portion to permit swinging of the swinging portion between non-gripping and gripping positions as it shifted from its retracted to its extended positions.

A still further aspect of the disclosure is the provision of such an assembly in which a gripping arm has substantial width at its gripping end to provide firm gripping of objects, and also to permit sweeping action with the gripping arm.

Yet another aspect of the disclosure is to provide a tool assembly adapted to break hardened materials and to manipulate materials whereby such joint operations can be achieved without placing workmen in unduly hazardous conditions.

These and other aspects of the disclosure will become more clearly apparent as the following description is read in conjunction with the drawings.

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BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a bottom front perspective view of an impact hammer having a gripping arm assembly attached thereto, with the gripping arms in retracted, non-gripping positions.

FIG. 2 is a view similar to FIG. 1, with the gripping arms in extended, gripping positions.

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FIG. 3 is an enlarged, exploded perspective view of a gripping arm removed from the assembly with associated mounting plates.

FIG. 4 is a front elevational view of the device illustrated in FIGS. 1 and 2, with the arms shown in various operating positions.

DESCRIPTION OF PREFERRED EMBODIMENT

Referring to FIGS. 1 and 2, at 10 is indicated generally apparatus according to an embodiment of the disclosure. A fluid-pressure-actuated impact hammer 12 is secured to and supported between a pair of opposed, laterally spaced support plates 14, 16. The upper ends of support plates 14, 16 are interconnected by a crossmember 18 and a shaft 20. Impact hammer 12 has an elongate impact member 20 with a lower, or distal, working end 20a. The impact hammer has a longitudinally extending impact axis 21, which extends through impact member 20. As is known, the impact hammer is operable to rapidly reciprocate member 20 under power to break, or separate, hardened material.

The outer, or distal, end of an elongate, maneuverable boom 22 is connected through various connecting members, such as links 24, 26 and other parts unseen in the figures to support plates 14, 16 permitting articulation of the impact hammer at the outer end of boom 22. An elongate extensible-retractable ram 28 is operatively connected to the support assembly for impact hammer 12, such that extension and retraction of the ram pivots the impact hammer relative to boom 22.

Boom 22 may be mounted on either stationary or mobile equipment. For example, a stationary base may be mounted adjacent an area in which the impact

hammer and mechanism is desired to be used, or it may be mounted on the boom of a vehicle allowing it to be moved to and used in a variety of locations.

Gripping assembly mechanism, indicated generally at 30, is secured to and moveable with impact hammer 12. Gripping assembly 30 includes a pair of opposed, laterally spaced apart, elongate gripping arms 32, 34. The gripping arms are attached to the impact hammer through mounting mechanism indicated generally at 36, 38.

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Gripper arms 32, 34, and mounting mechanism 36, 38 are substantially mirror images of each other mounted on opposite sides of impact hammer 12. Thus, only one set of such will be described in detail, recognizing that the other set is generally the same, but mounted in mirror-image fashion relative to the one described.

Referring to FIG. 3, gripping arm 34 includes an upper, or first, plate portion 40 and a lower, or second, plate portion 42. A hinge 44 pivotally interconnects plate portions 40, 42.

Plate portion 40 has a first pair of spaced apart projections 40a, 40b, also referred to as connector guides, extending outwardly from one of its side edges, and another pair of spaced apart projections 40c, 40d extending outwardly from its opposite side edge. Similar projections 42a, 42b, also referred to as swing guides, extend outwardly from opposite side edges of plate portion 42 in a region spaced from hinge 44. As seen in FIG. 3, projections 42a, 42b are substantially aligned with each other on opposite sides of plate portion 42. The distal, or lower, end portion 42c of plate portion 42 is referred to herein as a gripping portion. A gripping pad 48 is secured to the gripping portion of plate 42. The gripping pad 48 may be removably secured to plate 42, as by screws or bolts, permitting replacement as needed.

Referring to FIGS. 1, 2 and 4, the mounting mechanism 38 includes a backing plate 52. Backing plate 52 is bent into a configuration to conform to the bend lines of its associated support plate 16 of the impact hammer and has an upstanding upper portion 52a. The backing plate is secured to its associated support plate (14 or 16) as by bolts or screws 54.

A pair of guide plates 56, 58 have one set of edges 56a, 58a (see Fig. 3) configured to conform to the bends of plate 52 and are secured to plate 52, as by welding, along their formed edges 56a, 58a in substantially parallel, spaced-apart orientation as illustrated in FIGS. 1 and 2. The lateral spacing between the inner side surfaces of plates 56, 58 is slightly greater than the side-to-side width dimension noted W in FIG. 3 of gripping arm 34. Preferably width W may be in a range of 5 to 20 inches.

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Guide plates 56, 58 have guide tracks, or grooves, 62, 64 extending along their inwardly facing surfaces. Grooves 62, 64 are substantially parallel to each other, and thus only groove 64 will be described in detail, understanding that groove 62 would be similar. Groove 64 has an elongate first portion 64a extending substantially longitudinally of plate 58, a second portion 64b, which on extending downwardly is angled inwardly toward impact axis 21 at a small angle, and a third, or lower, portion 64c which is angled inwardly toward edge 56a and impact axis 21 at a greater angle than second portion 64b.

Projections 40a, 40b, 42a rest slidably in groove 64, and projections 40c, 40d, 42b rest slidably in groove 62.

Operator mechanism, in the form of a pair of elongate, fluid-actuated, extensible-retractable rams 68, 70 are operatively coupled to gripping arms 32, 34. The upper end of each of rams 68, 70 is secured to its associated upstanding portion 52a of a backing plate 52. The rod ends 68a, 70a of rams 68, 70 are connected to the first portions 40 of their associated gripping arms through pins 72 extending through holes 74 in the upper portions of the gripping arms. The rams may be supplied pressurized fluid from the same source which supplies impact hammer 12, or a separate source may be provided.

When rams 68, 70 are in their retracted positions, as illustrated in FIG. 1, portions 40, 42 of the gripping arms are substantially aligned in vertically disposed positions and are spaced upwardly and away from opposite sides of the operating tip 20a of impact member 20.

When rams 68, 70 are extended, gripping arms 32, 34 are shifted downwardly between their associated side guide plates. So long as the projections,

such as those indicated at 42a, 42b on gripping arm portion 42 remain in the vertically disposed portions of the guide channels, such as that indicated at 64a, gripping arm portion 42 does not begin to swing toward impact member 20. This somewhat extended, but still vertical, position for arms 32, 34 is indicated generally at 32a, 34a in dashed outline in FIG. 4.

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As rams 68, 70 continue to extend, projections such as those indicated at 42a, 42b move into the slightly inwardly angled second portions of the groove, such as that indicated at 64b, and the lower portions of the gripping arms begin to swing inwardly toward the impact axis 21 as indicated at 32b, 34b in FIG. 4.

As rams 68, 70 continue to extend and shift the gripping arms to their substantially fully extended positions, as indicated at 32c, 34c in FIG. 4, the gripping pads 48 on the opposed arms are swung together on opposite sides of the impact axis 21 in a region spaced longitudinally outwardly from the tip 20a of impact member 20. The spacing between projections on the upper, or first, portion 40 of a gripping arm is sufficiently above projections on the lower plate portion, such as those indicated at 42a, 42b, that the upper plate portion will remain substantially vertical in the upright groove portions such as that indicated at 64a, while the lower plate portions swing toward the impact axis 21.

The arms being swung toward each other under the power of rams 68, 70 allows the arms to grip material in the region of the impact hammer and manipulate it as desired.

Explaining operation of the apparatus thus described, with the gripping arms 32, 34 in their retracted position as illustrated in FIG. 1, impact member 20 of impact hammer 12 may be used in its normal fashion to break hardened material, such as rock, concrete, etc. By being mounted at the outer end of a moveable boom 22 and being manipulatable by action of ram 28 and other movement of boom 22, the impact hammer may be directed as desired.

Should it be desired to grip and move material in the region of the working end 20a of impact member 20, gripping arms 32, 34 may be extended by extension of rams 68, 70 to the gripping position illustrated in FIG. 2 and in solid outline in FIG. 4. The travel pattern for the second, or lower, portions of arms 32, 34 is

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indicated in FIG. 4, with the initial extension being as shown at 32a, 34a. As the arms are extended further and the projections thereon move slidably into angled portions of the tracks, such as that illustrated at 64b in FIG. 3, the arms swing in a slight amount as indicated in 32b, 34b. Extension to the full lower limit, as illustrated in FIG. 2 and solid outline in FIG. 4, causes the lower ends of arms 32, 34 to swing together to grip material therebetween. Gripping pads 48 assist in gripping.

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With material gripped between arms 32, 34, the boom 22 and the mount for impact hammer 20 may be manipulated to lift and move gripped material. Upon movement to a selected position, it is a simple matter to retract rams 68, 70 to swing the gripping arms open, thus releasing the grip on such material and releasing it from the arms.

Such equipment should work well in mining operations where it may be necessary to clear material from screens, such as grizzlies. In such operation, the boom may maneuver impact member 20 to break up large boulders or other materials on the screen, or grizzly, such that it may be sized to fall through the screen. Should debris which should not go through the grizzly be noted in the material, such debris may be grasped by arms 32, 34 and removed from the grizzly.

Further, with the arms in their extended positions, as illustrated in FIG. 2, it will be seen that they have substantial width and may be used in a side-to-side sweeping action upon movement of boom 22 in a side-to-side fashion to move material about in the mass of material on the grizzly. Also, with the arms retracted the mounting plates (56, 58) have sufficient width that they may be used in a fore-to-aft sweeping action.

While a preferred embodiment has been described herein, it should be apparent to those skilled in the art that variations and modifications are possible without departing from the spirit of the invention.

We claim:

1. A gripping assembly adapted to be mounted on an impact hammer having a longitudinally extending impact axis, the assembly comprising

a pair of opposed, laterally spaced apart, elongate gripping arms, each of which has a gripping portion adjacent one of its ends,

mounting mechanism adapted to mount said arms on said impact hammer with said gripping arms spaced to opposite sides of said impact axis and shiftable longitudinally between retracted and extended position, with said gripping portions of said arms spaced apart a selected distance when said arms are retracted and said gripping portions moving toward each other as said gripping arms are shifted toward their extended positions, and

operator mechanism coupled to said gripping arms operable to shift said arms between their retracted and extended positions.

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2. The assembly of claim 1, wherein said mounting mechanism comprises a track to which a gripping arm is coupled, with said track guiding movement of said gripping portion as said gripping arm is shifted between its retracted and extended positions.

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3. The assembly of claim 2, wherein said mounting mechanism comprises a plate element having an elongate groove formed therein defining said track and said gripping arm has a projection thereon slidably received for movement in said groove.

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4. The assembly of claim 1, wherein a gripping arm comprises a connector portion connected to said operator mechanism and a swinging portion pivotally coupled to said connector portion to permit swinging of said swinging portion relative to said connector portion toward and away from said impact axis.

5. The assembly of claim 4, wherein said mounting mechanism comprises an elongate track, a connector guide on said connector portion engaging and guided by said track, and a swing guide on said swinging portion engaging and guided by said track, with a first guide portion of said track extending substantially parallel to said impact axis and a second guide portion of said track angled toward said impact axis, the positions of said first and second guide portions being such that said connector portion and said swinging portion remain spaced from said impact axis throughout a major portion of their travel from their retracted toward their extended positions, and upon nearing the extended position said swinging portion swings toward said impact axis.

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- 6. The assembly of claim 5, wherein said gripping portion is on a distal end portion of said swinging portion.
- 7. The assembly of claim 1, wherein said operator mechanism comprises an elongate fluid-actuated ram.
 - 8. The assembly of claim 1, wherein a gripping pad is mounted on a distal gripping end portion of said gripping arm.

9. The assembly of claim 1, wherein the impact hammer has an elongate impact member disposed on said impact axis and a gripping arm comprises an elongate plate element which when in its extended position is disposed adjacent said impact member.

10. The assembly of claim 9, wherein said gripping arm has substantial width at its gripping end to permit sweeping action.

11. A tool assembly adapted to break hardened materials and manipulate30 materials comprising

an impact hammer having a material impacting member disposed on a longitudinally extending impact axis,

a pair of opposed, laterally spaced apart, elongate gripping arms, each of which has a gripping portion adjacent one of its ends,

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mounting mechanism mounting said arms on said impact hammer with said gripping arms spaced to opposite sides of said impact axis and shiftable longitudinally between retracted and extended position, with said gripping portions of said arms spaced apart a selected distance when said arms are retracted, and said gripping portions moving toward each other as said gripping arms are shifted toward their extended positions, and

operator mechanism coupled to said gripping arms operable to shift said arms between their retracted and extended positions.

- 12. The assembly of claim 11, wherein said mounting mechanism comprises a track to which a gripping arm is coupled, with said track guiding movement of said gripping portion as said gripping arm is shifted between its retracted and extended positions.
- 13. The assembly of claim 12, wherein said mounting mechanism
 20 comprises a plate element having an elongate groove formed therein defining said
 track and said gripping arm has a projection thereon received for sliding movement
 in said groove.
- 14. The assembly of claim 11, wherein a gripping arm comprises a connector portion connected to said operator mechanism and a swinging portion pivotally coupled to said connector portion to permit swinging of said swinging portion relative to said connector portion toward and away from said impact axis.
- 15. The assembly of claim 14, wherein said mounting mechanism comprises an elongate track, a connector guide on said connector portion engaging and guided by said track, and a swing guide on said swinging portion engaging and

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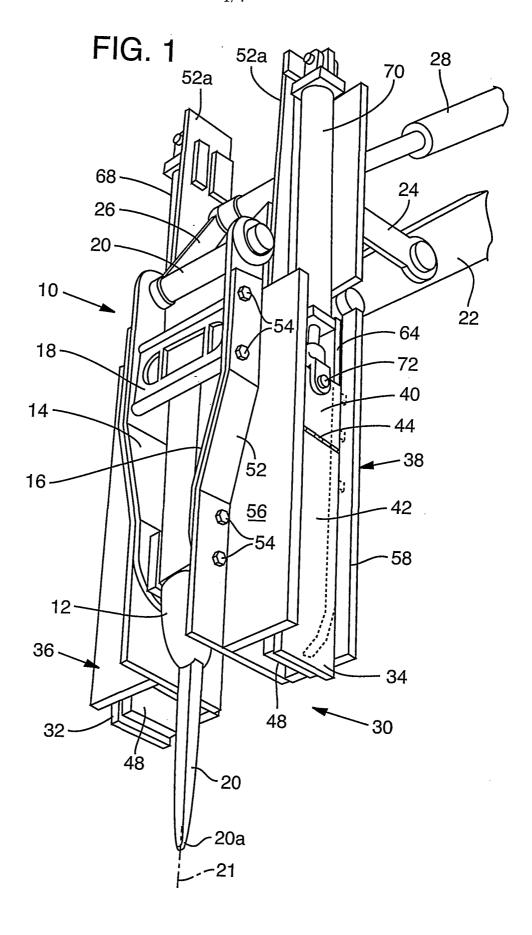
guided by said track, with a first guide portion of said track extending substantially parallel to said impact axis and a second guide portion of said track angled toward said impact axis, the positions of said first and second guide portions being such that said connector portion and said swinging portion remain spaced from said impact axis throughout a major portion of their travel from their retracted toward their extended positions, and upon nearing the extended position said swinging portion swings toward said impact axis.

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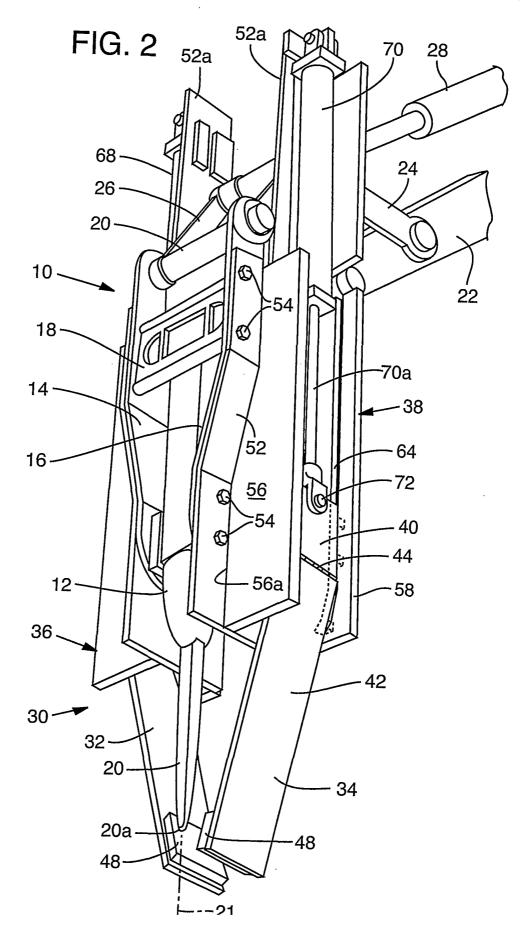
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- 16. The assembly of claim 15, wherein said gripping portion is on a distal end portion of said swinging portion.
 - 17. The assembly of claim 11, wherein said operator mechanism comprises an elongate fluid-actuated ram.
- 15 18. The assembly of claim 11, wherein a gripping pad is mounted on a distal gripping end portion of said gripping arm.
 - 19. The assembly of claim 11, wherein a gripping arm comprises an elongate plate element which when in its extended position is disposed adjacent said impact member.
 - 20. The assembly of claim 19, wherein said gripping arm has substantial width at its gripping end to permit sweeping action.
- 25 21. The assembly of claim 11, which further comprises means adapted to mount said assembly on a movable boom.
- 22. The assembly of claim 11, wherein said impact member has a distal material engaging end and said gripping portions of said arms, when said arms are shifted to their extended positions, are configured to close on said impact axis in a region spaced longitudinally outwardly from said distal end of said impact member.

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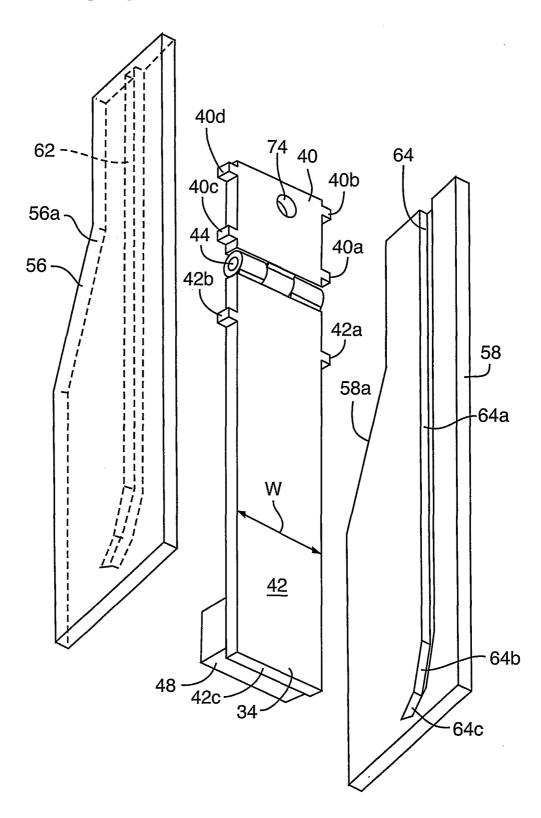


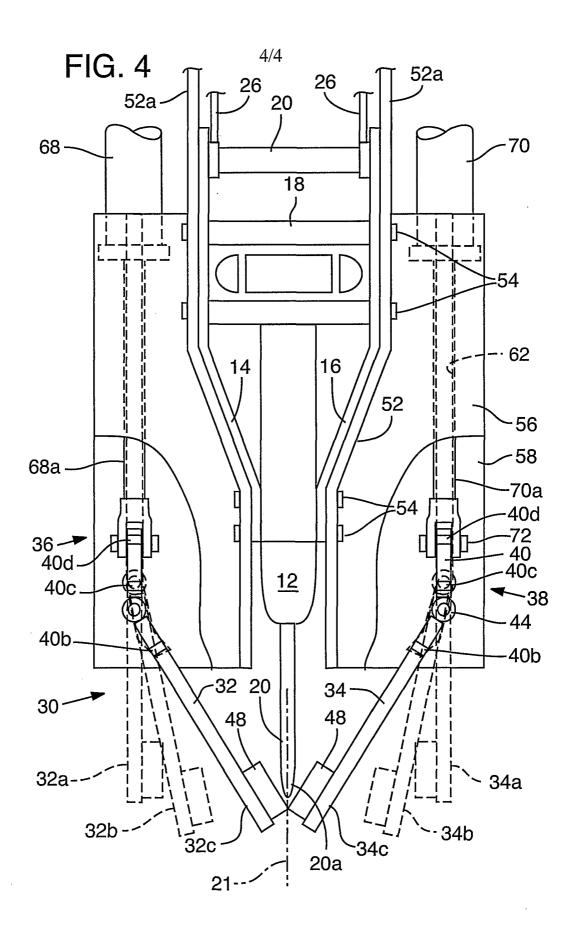
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FIG. 3





INTERNATIONAL SEARCH REPORT

Intel II Application No PCT/US 02/10819

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A. CLASSI IPC 7	FICATION OF SUBJECT MATTER B25D17/00				
According to	o International Patent Classification (IPC) or to both national classif	ication and IPC			
	SEARCHED				
Minimum do IPC 7	ocumentation searched (classification system followed by classification $B25J - B25D$	ation symbols)			
Documenta	tion searched other than minimum documentation to the extent that $$	such documents are included	in the fields searched		
	lata base consulted during the international search (name of data be ternal, WPI Data, PAJ	pase and, where practical, sear	ch terms used)		
C. DOCUM	ENTS CONSIDERED TO BE RELEVANT				
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<u> </u>	her documents are listed in the continuation of box C.	X Patent family mem	pers are listed in annex.		
"A" docume consic "E" earlier of filing c "L" docume which citation "O" docume other i "P" docume later th	ent defining the general state of the art which is not letered to be of particular relevance document but published on or after the international late ent which may throw doubts on priority claim(s) or is cited to establish the publication date of another or other special reason (as specified) ent referring to an oral disclosure, use, exhibition or means ent published prior to the international filing date but no actual completion of the international search	"T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention "X" document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone "Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such document is combined with one or more other such documents, such combination being obvious to a person skilled in the art. "&" document member of the same patent family Date of mailing of the international search report			
	5 July 2002		24/07/2002		
Name and r	nailing address of the ISA European Patent Office, P.B. 5818 Patentlaan 2 NL – 2280 HV Rijswijk Tel. (+31–70) 340–2040, Tx. 31 651 epo nl, Fax: (+31–70) 340–3016	Authorized officer Fiorani, G			

INTERNATIONAL SEARCH REPORT

Information on patent family members

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