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(54) Title: ANNULAR INTERCOOLER HAVING CURVED FINS

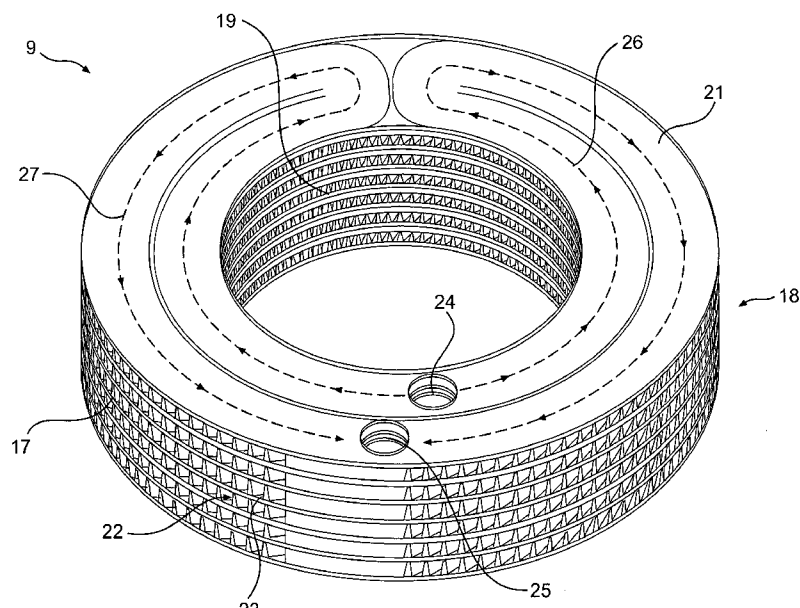


FIG. 3

(57) Abstract: An air-to-fluid intercooler (9) is disclosed. The air-to-fluid intercooler may include a core assembly (18) including an outer circumference (17) and an inner circumference (19), at least one annular tube body (21) configured to direct flow of a cooling fluid within the core assembly, and at least one curved fin (22) coupled to an exterior surface (23) of the at least one annular tube body and configured to direct a flow of charge air (5b) through the core assembly.

WO 2009/073124 A1

Description

ANNULAR INTERCOOLER HAVING CURVED FINS

Technical Field

The present disclosure is directed to an intercooler and, more
5 particularly, an annular intercooler having curved fins.

Background

Construction and earthmoving equipment, as well as many other
types of machines, are commonly used in a wide variety of applications.
Generally, a machine is powered by an internal combustion engine. In order to
10 enhance the performance of the machine, the engine must perform as efficiently
as possible. Because many machines are powered by internal combustion
engines, various methods have been developed to increase internal combustion
engine efficiency. One method has been to incorporate a two-stage or twin-
compressor turbocharger into the internal combustion engine. The turbocharger
15 may compress air prior to entering an engine intake or combustion chamber.
Supplying the engine intake with compressed air ("charge air") may allow for
more complete combustion. This may result in improved power density and
better engine efficiency. However, compressing the air may also cause an
increase in the intake air temperature. Supplying the engine intake with such
20 heated charge air may lead to an undesirable increase in the amount of emissions
exiting from the engine. Also, because engines generally produce large quantities
of heat already, adding heated charge air to the engine intake or combustion
chamber may increase the operating temperature of the engine, thus resulting in
excessive wear on engine components.

25 An air-to-fluid intercooler may be disposed between a first stage
compressor and a second stage compressor of the twin-compressor turbocharger
and be used to reduce smoke and other engine emissions, such as, nitrous oxides,

-2-

by cooling the charge air from the first stage compressor before it enters the second stage compressor and, ultimately, the engine intake manifold. Using the air-to-fluid intercooler may also result in lower combustion temperatures, thus improving engine component life by reducing thermal stress on the engine and
5 increasing engine power output. Also, disposing the intercooler between the first stage compressor and second stage compressor may increase the component life and efficiency of the second stage compressor.

The air-to-fluid intercooler may include one or more passages configured to direct flow of heated charge air. The passages may contain heat
10 transfer enhancements, such as, one or more fins, dimples or other surface modifications. The passages may be coupled to an annular tube body configured to direct flow of some type of cooling fluid, for example, a liquid coolant, which may cool the passage. As the heated charge air passes through the passage, it may come into contact with the heat transfer enhancements, such as, fins, of the
15 passage body. Heat may be transferred from the charge air to the fins of the passage, and then from the fins into the liquid coolant, thus removing heat from the charge air.

The flow of charge air from the first stage compressor may enter the air-to-fluid intercooler in a swirling motion. As the flow of charge air enters
20 the passages of the air-to-fluid intercooler, the fins of the passages may disrupt and change the flow direction of the charge air, resulting in a pressure drop of the charge air and ultimately poor engine performance. Also, radial and axial space and size limitations may exist because the air-to-fluid intercooler may be disposed between the first stage compressor and the second stage compressor.
25 Therefore, the heated charge air may flow in the passage of the air-to-fluid intercooler for an inadequate duration resulting in poor heat exchange performance.

One method of improving the performance of an air-to-fluid intercooler under such conditions is described in U.S. Patent No. 7,278,472 (the

'472 patent) to Meshenky et al., issued on 9 October 2007. The '472 patent describes a heat exchanger used as an intercooler in a combustion air charging device such as a turbocharger or a supercharger. The heat exchanger is disposed in a housing and between a compressor wheel and an outlet. The heat exchanger
5 has a donut-shaped core. The core includes a gas flow path with a substantial radial extent and a gas inlet in fluid communication with the compressor wheel and a gas outlet in fluid communication with the housing outlet. A coolant flow path is provided in the intercooler in heat exchange with the gas flow path and has a substantial axial extent. Flattened tubes are also employed in the core and
10 are arranged tangential to a circle concentric with the center for the core. The canting of the flattened tubes is against the direction of swirling charge air flow entering the intercooler and provides a smoother transition of the air flow into the spaces between the tubes to minimize turning losses of the air flow, and thereby reduce pressure loss.

15 Although the intercooler of the '472 patent may improve performance of an air-to-fluid intercooler, it may have limitations. For example, because of the canted configuration of the flattened tubes, the swirling flow of charge air will still be disrupted. The passages created by the flattened tubes will not accommodate the swirling direction of the charge air as air flow and direction
20 would be interrupted by the tangential walls of the tubes, ultimately resulting in a pressure loss. Also, the passages formed by the tubes do not create adequate air flow length for charge air to flow and undergo cooling.

The air-to-fluid intercooler of the present disclosure is directed towards improvements in the existing technology.

25 Summary of the Disclosure

One aspect of the present disclosure is directed to an air-to-fluid intercooler. The air-to-fluid intercooler may include a core assembly including an outer circumference and an inner circumference, at least one annular tube body configured to direct flow of a cooling fluid within the core assembly, and at

-4-

least one curved fin coupled to an exterior surface of the at least one annular tube body and configured to direct a flow of charge air through the core assembly.

Another aspect of the present disclosure is directed to a method for cooling compressed charge air with an air-to-fluid intercooler, the air-to-fluid
5 intercooler including a core assembly, an outer circumference, and an inner circumference. The method may include directing a flow of cooling fluid to the core assembly through at least one annular tube body and directing a flow of compressed charge air with at least one curved fin coupled to an exterior surface of the at least one annular tube body.

10 Brief Description of the Drawings

Fig. 1 is a diagrammatic illustration of a machine provided with an air-to-fluid intercooler according to an exemplary disclosed embodiment;

Fig. 2 is a diagrammatic view of an air-to-fluid intercooler employed in a twin-compressor turbocharger according to an exemplary
15 disclosed embodiment;

Fig. 3 is a diagrammatic view of an air-to-fluid intercooler for a machine according to an exemplary disclosed embodiment; and

Fig. 4 is a sectional view of an air-to-fluid intercooler for a machine according to an exemplary disclosed embodiment.

20 Detailed Description

Fig. 1 diagrammatically illustrates an exemplary machine 10. The outline 2 represents a portion of the chassis of the machine 10. An engine 1 may be mounted on the chassis 2. Engine 1 may be any type of engine, for example, an internal combustion engine of the gas, diesel, and/or gaseous fuel type. In the
25 illustrative example, engine 1 is shown with six combustion chambers 3a-3f for generating power, each provided with a piston, one or more intake valves, one or more exhaust valves, and other components (not shown) known to those having skill in the art. Engine 1 may include a turbocharger 4 for compressing intake air

5a to form compressed charge air. Turbocharger 4 may be a twin-compressor turbocharger including a first stage compressor 7 and a second stage compressor 8. Due to the heat of compression from first stage compressor 7, the compressed charge air exits first stage compressor 7 as heated charge air 5b and is directed to
5 an air-to-fluid intercooler 9, where the fluid may be a cooling fluid, such as, a liquid coolant or a gaseous fluid. Air-to-fluid intercooler 9 cools heated charge air 5b prior to entering second stage compressor 8 and ultimately an air intake manifold 6. In this embodiment one turbocharger 4 with a first stage compressor 7 and a second stage compressor 8 is illustrated, but it will be understood that the
10 number of turbochargers or the number of compressors could be one or more than one and still fall within the scope of this disclosure. Furthermore, expedients other than turbochargers, such as an engine driven supercharger or superchargers, may be employed to compress intake air 5a, forming heated charge air 5b.

First stage compressor 7 and second stage compressor 8 may be
15 powered by a turbine 11 driven by engine exhaust flow 12. First stage compressor 7 may pressurize intake air 5a to allow a greater mass of fuel/air mixture in the engine cylinders of engine 1. The result may be an increase in power and improved engine efficiency. However, as a byproduct of pressurization, the temperature of intake air 5a may also increase, which may be
20 undesirable. The compressed intake air exiting first stage compressor 7 may conveniently be referred to as heated charge air 5b. As noted above, heated charge air 5b may be cooled prior to entering second stage compressor 8 by passing through air-to-fluid intercooler 9. Air-to-fluid intercooler 9 may be provided downstream of first stage compressor 7 and upstream of second stage
25 compressor 8 and air intake manifold 6. Once cooled charge air 5c exits air-to-fluid intercooler 9, it undergoes a second compression event by second stage compressor 8 and exits as combustion air 5d before entering air intake manifold 6 to further increase the mass of the fuel/air mixture in the combustion chambers 3a-3f. Although not shown, it will be understood by those skilled in the art that

combustion air 5d may exit second stage compressor 8 of turbocharger 4 and enter another heat exchanger, such as, an air-to-air aftercooler, prior to entering air intake manifold 6.

Fig. 2 illustrates an exemplary embodiment of how air-to-fluid intercooler 9 may be configured. Turbocharger 4 may include a housing 13 configured to enclose first stage compressor 7 and second stage compressor 8 of rotary shaft 14 and air-to-fluid intercooler 9 operatively connected between first stage compressor 7 and second stage compressor 8. First stage compressor 7 and second stage compressor 8 may embody wheels, as shown, or other suitable expedients. Rotary shaft 14 may also include a turbine wheel 11 configured to be driven by engine exhaust flow 12 and to subsequently drive rotary shaft 14.

Housing 13 may also include an intake air inlet 15 configured to direct intake air 5a to the inlet side of first stage compressor 7 of turbocharger 4. A combustion air outlet 20, on the outlet side of second stage compressor 8, may be configured to expel combustion air 5d.

Intake air 5a may enter intake air inlet 15 and flow to the inlet side of first stage compressor 7. As exhaust flow 12 drives turbine wheel 11 and rotary shaft 14, first stage compressor 7 may compress intake air 5a to heated charge air 5b. Heated charge air 5b may exit first stage compressor 7 radially outward and in a swirling motion. Heated charge air 5b may continue to flow through space 16 of housing 13 and enter an outer circumference 17 of air-to-fluid intercooler 9. Heated charge air 5b may then pass through a core assembly 18 of air-to-fluid intercooler 9 in a radially inward direction and be cooled by cooling fluid that flows through core assembly 18, which will be explained in later embodiments. Once cooled, heated charge air 5b may exit an inner circumference 19 of air-to-fluid intercooler 9 as cooled charge air 5c and flow towards the inlet side of second stage compressor 8 in a swirling motion, preserving the kinetic energy from the first stage compression. Cooled charge air 5c then undergoes further compression and exits as combustion air 5d.

-7-

Combustion air 5d is then discharged from turbocharger 4 through combustion air outlet 20 and to air intake manifold 6 as shown in Fig. 1.

An exemplary embodiment of air-to-fluid intercooler 9 will be described in greater detail as illustrated in Fig. 3 and Fig. 4. Core assembly 18 of
5 air-to-fluid intercooler 9 may include an outer circumference 17 and an inner circumference 19, as discussed in the previous embodiment of Fig. 2. Air-to-fluid intercooler 9 may also include at least one cooling fluid passage, such as, annular tube body 21, and at least one curved fin 22. Curved fin 22 may be coupled to exterior surface 23 of annular tube body 21 and may be configured to
10 direct flow of heated charge air 5b. Core assembly 18 may also include a plurality of annular tube bodies 21 and curved fins 22 axially assembled. An inlet tube 24 and an outlet tube 25 may be associated with core assembly 18. Inlet tube 24 may be configured to direct a cooling fluid into annular tube body 21, and outlet tube 25 may be configured to direct the cooling fluid out of annular
15 tube body 21.

As shown in Fig. 3, an inlet cooling fluid flow 26 from inlet tube 24 may split and pass in a radially inner direction of annular tube body 21. Inlet cooling fluid flow 26 may then reverse direction and become an outlet cooling fluid flow 27, passing in a radially outer direction of annular tube body 21 to
20 outlet tube 25. Therefore, inlet cooling fluid flow 26 may be directed to inner circumference 19 of air-to-fluid intercooler 9 and provide further cooling of heated charge air 5b that has already been cooled by the outlet cooling fluid flow 27 at the outer circumference 17. Although not shown, it will be understood that annular tube body 21 may be configured to direct cooling fluid in other flow
25 directions, such as, cross flow, parallel flow, multi-pass flow, and counter flow.

As shown in Fig. 4, curved fin 22 may form a radially extending arced member 28 including a first end 29 associated with outer circumference 17 of air-to-fluid intercooler 9 and a second end 30 associated with inner circumference 19 of air-to-fluid intercooler 9. Arced member 28 may also be

-8-

configured to curve in a direction with the flow of heated charge air 5b. An additional curved fin 22 may be coupled to external surface 23 of annular tube body 21 and form a passage 31 with an adjacent curved fin 22. Passage 31 may include a first aperture 32 associated with outer circumference 17 serving as an inlet for heated charge air 5b and a second aperture 33 associated with inner circumference 19 serving as an outlet for cooled charge air 5c. Although not shown in Fig. 3 and Fig. 4, it will be understood that expedients other than fins, such as, dimples or other surface enhancements having a curved configuration, may be employed to direct flow of heated charge air 5b and transfer heat from charge air to the cooling fluid.

Industrial Applicability

The disclosed exemplary air-to-fluid intercooler 9 may have applicability with internal combustion engines. In particular, and as shown in Fig. 1 and Fig. 2, air-to-fluid intercooler 9 may serve to cool a flow of heated charge air 5b exiting first stage compressor 7 of a turbocharger 4 before it enters second stage compressor 8 and ultimately air intake manifold 6 of an engine 1, thus decreasing the level of emissions, increasing the life of engine components, and increasing engine power density.

In a machine 10, exhaust flow 12 leaving engine 1 may be directed towards a turbine wheel 11 of turbocharger 4. The flow of exhaust 12 may power turbine wheel 11, causing rotary shaft 14 to rotate and drive first stage compressor 7 and second stage compressor 8. Intake air 5a may be directed into first stage compressor 7 where it may undergo compression, and as a byproduct of compression, intake air 5a may also be heated into heated charge air 5b. Heated charge air 5b may flow from first stage compressor 7 in a swirling motion and into outer circumference 17 of air-to-fluid intercooler 9. As shown in Fig. 3 and Fig. 4, heated charge air 5b may enter air-to-fluid intercooler 9 through first aperture 32 of passage 31. Arced members 28 forming curved fins 22 of passage 31 curve in the direction of heated charge air 5b flow. Therefore, curved fins 22

-9-

may direct and maintain the swirling flow of heated charge air 5b, preventing significant changes in flow direction and ultimately a pressure drop of heated charge air 5b. Curved fins 22 of passage 31 may be coupled to an annular tube body 21 configured to direct flow of some type of cooling fluid, for example, a liquid coolant, which may cool passage 31. As heated charge air 5b passes through passage 31, it may come into contact with curved fins 22 of passage 31. Heat may be transferred from heated charge air 5b to curved fins 22 of passage 31, and then from curved fins 22 into the liquid coolant, thus removing heat from heated charge air 5b. Passage 31 formed by curved fins 22 may have a longer passage than compared to a similar passage formed by fins having a straight configuration. Therefore, passage 31 may expose heated charge air 5b to cooling for a longer period, improving heat exchanging performance. Once cooled, heated charge air 5b may be discharged from inner circumference 19 of air-to-fluid intercooler 9 as cooled charge air 5c to second stage compressor 8 where it undergoes further compression. Curved fins 22 of passage 31 may maintain the swirling motion of heated charge air 5b, and subsequently, cooled charge air 5c exiting second aperture 33 of passage 31 associated with inner circumference 19 may continue to flow in the swirling motion. The maintained swirling flow of cooled charge air 5c may form a smooth transition into second stage compressor 8 and save rotational kinetic energy as second stage compressor 8 may rotate in the same direction as the swirling flow. Compressed, cooled charge air 5c may exit second stage compressor 8 as combustion air 5d and then be discharged from turbocharger 4 through combustion air outlet 20. Combustion air 5d may be mixed with fuel within one or more combustion chambers 3a-3f within engine 1. Because cooler air has greater density than heated air, a volume of combustion air 5d at a certain pressure may contain a greater number of air molecules than the same volume of heated charge air 5b at that same pressure. Increasing the number of air molecules in combustion chambers 3a-3f of engine may decrease the amount of smoke and/or emissions exiting from engine 1, improve engine

-10-

performance, and improve engine efficiency. Also, reducing the temperature of heated charge air 5b may decrease the operating temperature of engine 1, thus resulting in less wear on engine components.

Assembling curved fins 22 to core assembly 18 of air-to-fluid
5 intercooler 9 may enhance cooled charge air 5c entering second stage compressor 8 and ultimately combustion air 5d entering combustion chambers 3a-3f within engine 1. Curved fin 22 may maintain the swirling flow of heated charge air 5b entering air-to-fluid intercooler 9, preventing a pressure drop and decrease in density of cooled charge air 5c exiting air-to-fluid intercooler 9. Also, passage 31
10 formed by curved fins 22 may expose heated charge air 5b to a longer period of cooling, increasing the density and the number of air molecules of cooled charge air 5c.

It will be apparent to those skilled in the art that various
modifications and variations can be made to the air-to-fluid intercooler of the
15 present disclosure without departing from the scope of the disclosure. Other embodiments will be apparent to those skilled in the art from consideration of the specification and practice of the embodiments disclosed herein. It is intended that the specification and examples be considered as exemplary only, with a true scope of the disclosure being indicated by the following claims.

-11-

Claims

1. An air-to-fluid intercooler (9), comprising:
a core assembly (18) including an outer circumference (17) and an
5 inner circumference (19);
at least one annular tube body (21) configured to direct flow of a
cooling fluid within the core assembly; and
at least one curved fin (22) coupled to an exterior surface (23) of
the at least one annular tube body and configured to direct a flow of charge air
10 (5b) through the core assembly.
2. The air-to-fluid intercooler of claim 1, wherein the at least
one curved fin extends from a first end (29) associated with the outer
circumference to a second end (30) associated with the inner circumference.
15
3. The air-to-fluid intercooler of claim 2, wherein the curved
fin curves in the direction of the flow of charge air entering the intercooler.
4. The air-to-fluid intercooler of claim 3, wherein an
20 additional curved fin and an adjacent curved fin form a passage (31), the passage
including a first aperture (32) associated with the outer circumference and a
second aperture (33) associated with the inner circumference.
5. The air-to-fluid intercooler of claim 1, including an inlet
25 tube (24) associated with the core assembly and configured to direct cooling fluid
into the at least one annular tube body and an outlet tube (25) associated with the
core assembly and configured to direct cooling fluid out of the at least one
annular tube body.

-12-

6. A method for cooling compressed charge air (5b) with an air-to-fluid intercooler (9), the air-to-fluid intercooler including a core assembly (18), an outer circumference (17), and an inner circumference (19), the method comprising:

5 directing a flow of cooling fluid to the core assembly through at least one annular tube body (21); and

directing a flow of compressed charge air with at least one curved fin (22) coupled to an exterior surface (23) of the at least one annular tube body.

10 7. The method of claim 6, wherein directing the flow of compressed charge air includes directing the flow from a first end (29) of the curved fin adjacent to the outer circumference of the air-to-fluid intercooler to a second end (30) of the curved fin adjacent to the inner circumference of the air-to-fluid intercooler.

15 8. The method of claim 7, wherein directing the flow of compressed charge air includes directing the flow of compressed charge air in the same direction as the flow of compressed charge air entering the intercooler.

20 9. The method of claim 6, further including directing the flow of cooling fluid to the core assembly through an inlet tube (24) configured to direct cooling fluid into the at least one annular tube body and directing the flow of cooling fluid from the core assembly through an outlet tube (25) configured to direct cooling fluid out of the at least one annular tube body.

25 10. An engine assembly (1), comprising:
an engine air intake manifold (6);

-13-

a multi-compressor turbocharger (4) configured to compress charge air (5a) before it enters the engine air intake manifold, the multi-compressor turbocharger including:

5 a housing (13) including at least one intake air inlet (15) and at least one combustion air outlet (20);

a rotary shaft (14) extending through the housing, the rotary shaft including at least two compressors (7, 8) and a turbine wheel (11) mounted thereon; and

10 an air-to-fluid intercooler as in any one of claims 1-5 operatively connected between the at least two compressors.

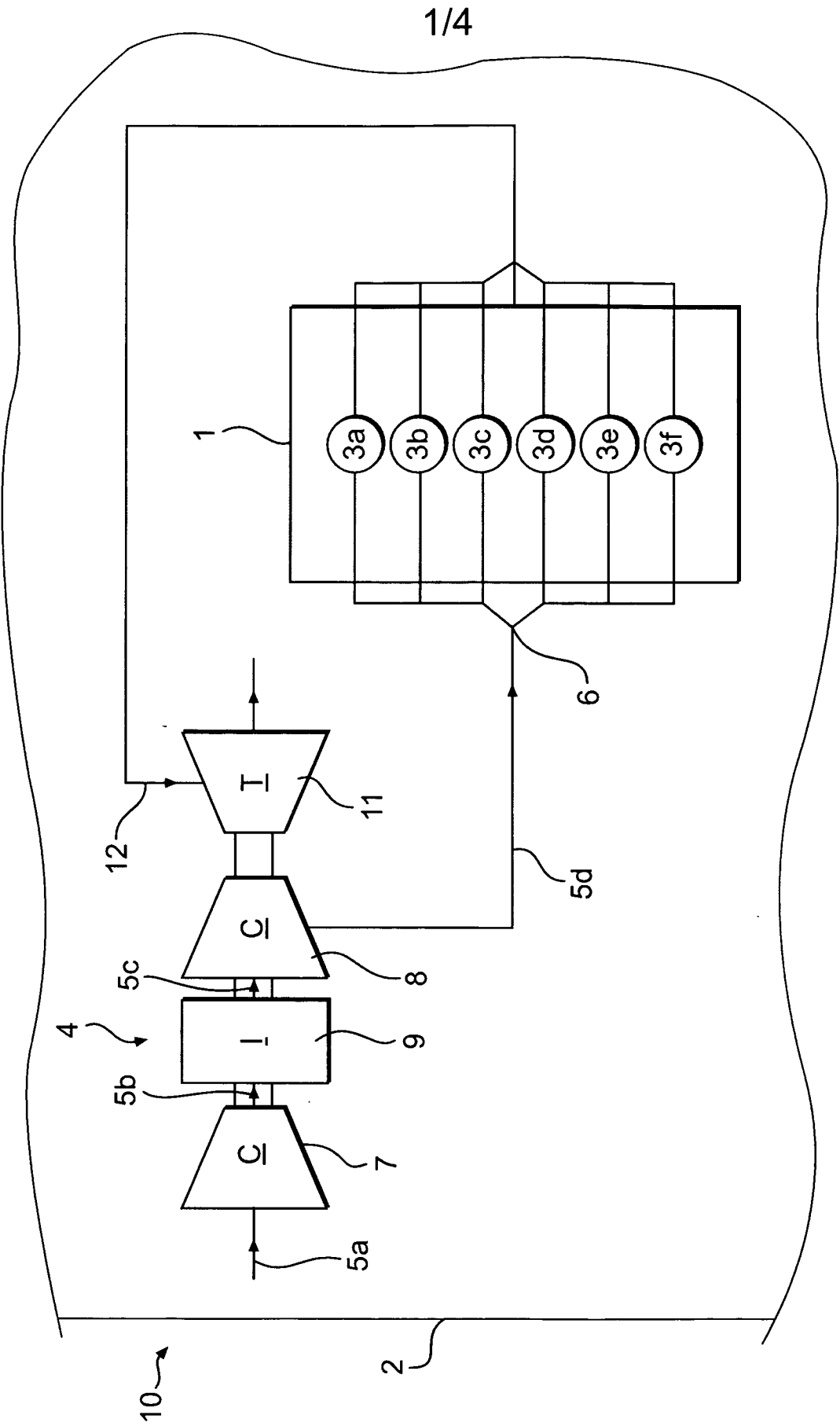
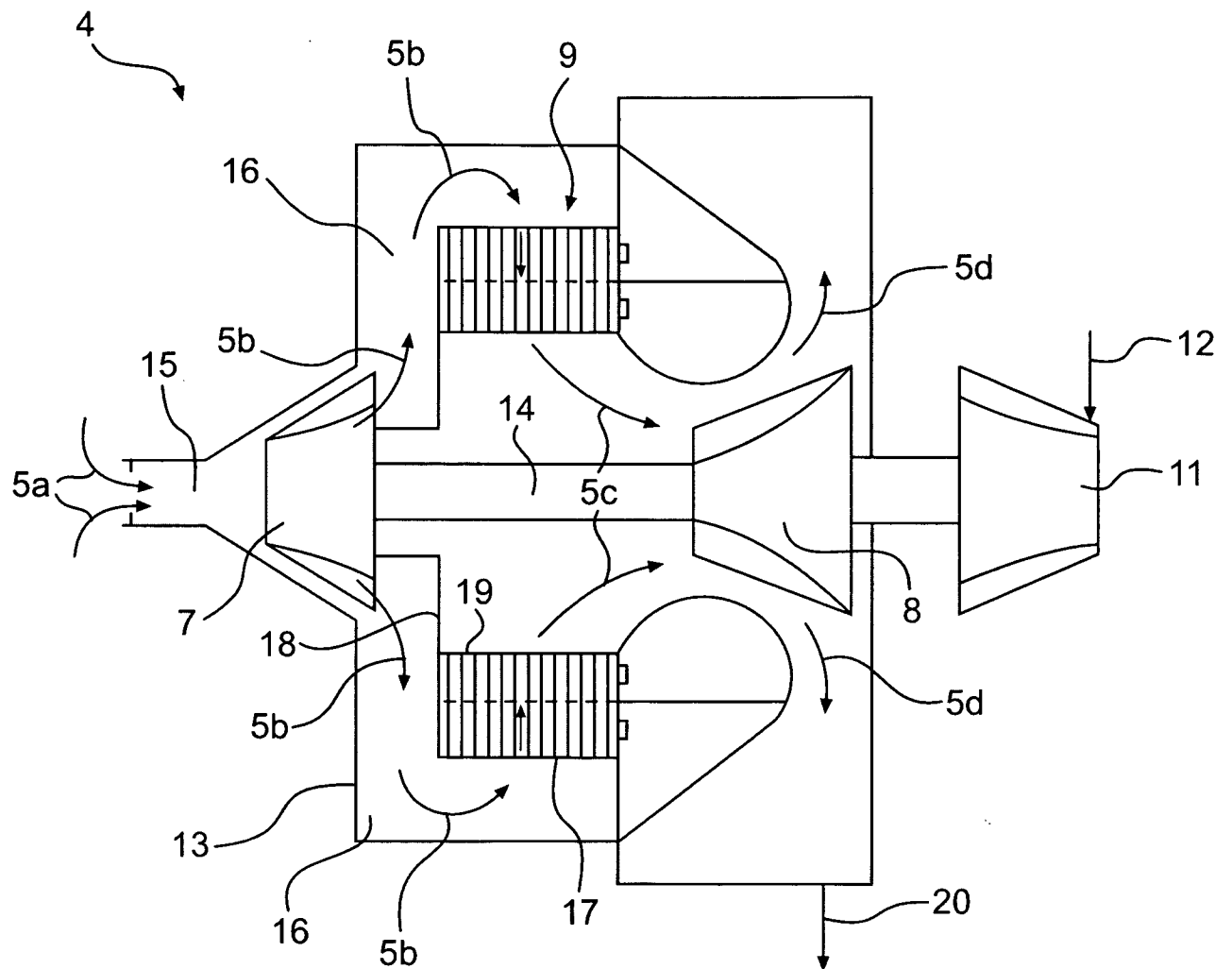
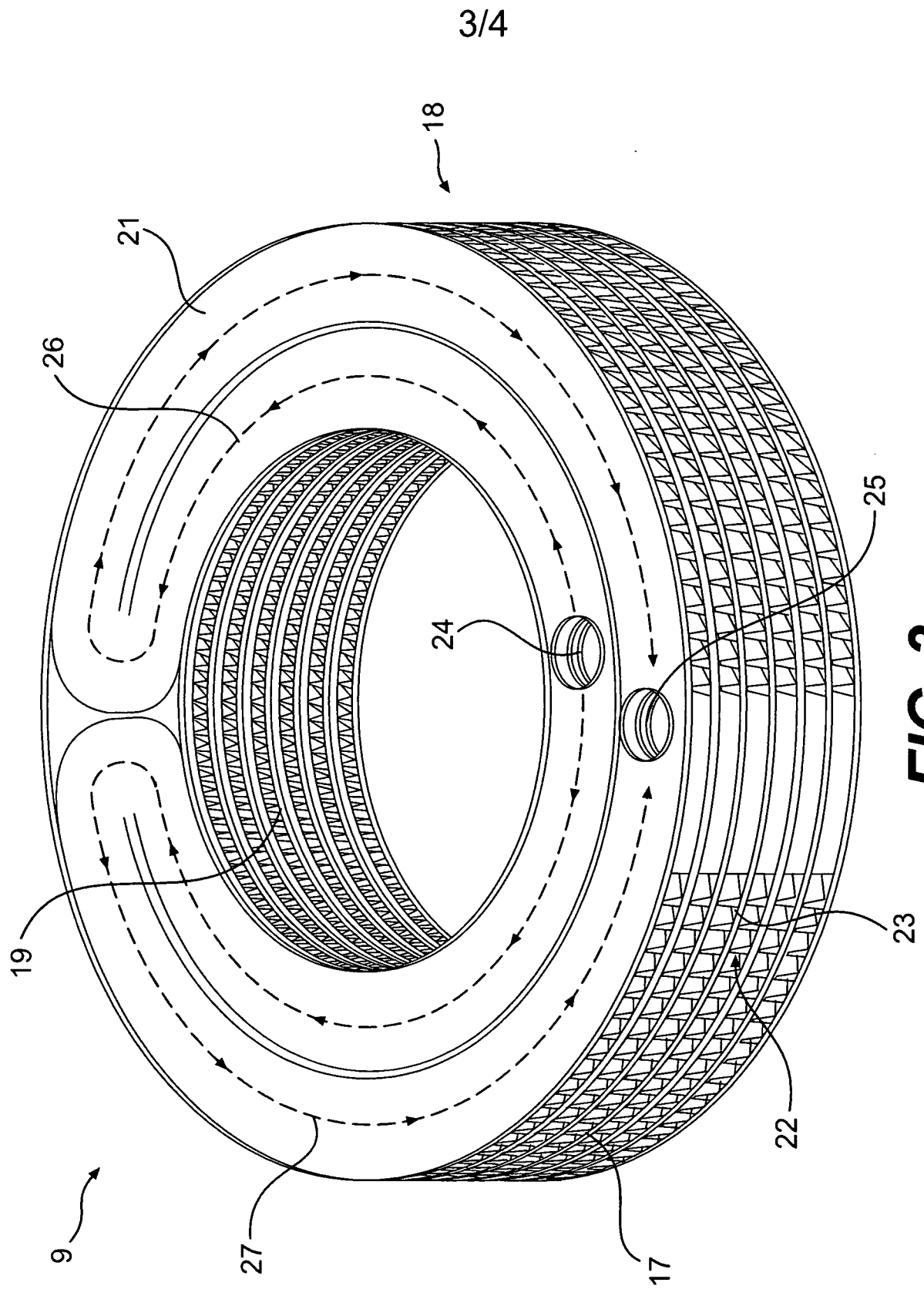


FIG. 1

2/4

**FIG. 2**



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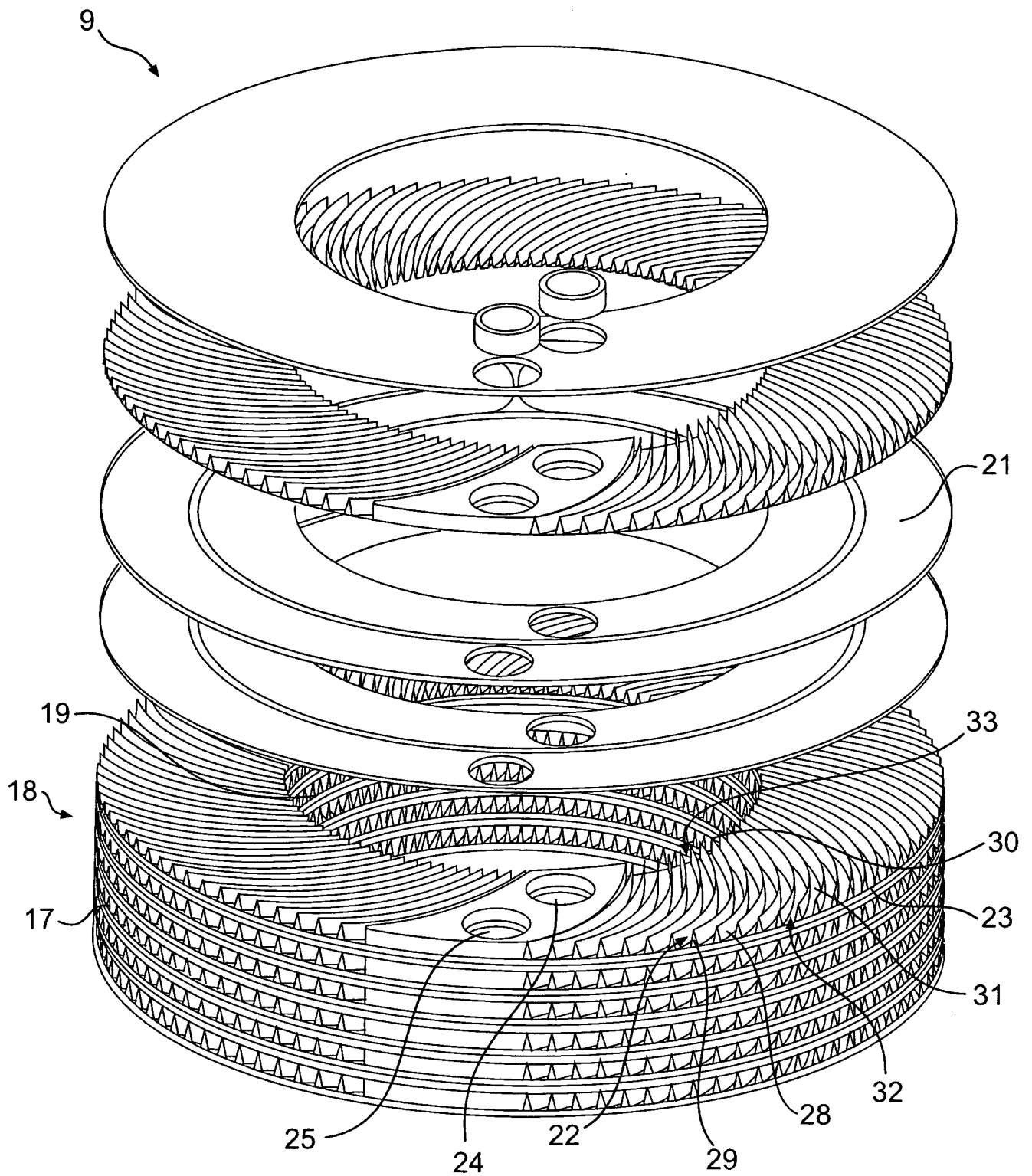


FIG. 4

INTERNATIONAL SEARCH REPORT

International application No
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A. CLASSIFICATION OF SUBJECT MATTER INV. F02B29/04 F28D7/00 F28F1/02 F28F1/34		
According to International Patent Classification (IPC) or to both national classification and IPC		
B. FIELDS SEARCHED Minimum documentation searched (classification system followed by classification symbols) F02B F28D F28F		
Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched		
Electronic data base consulted during the international search (name of data base and, where practical, search terms used) EPO-Internal		
C. DOCUMENTS CONSIDERED TO BE RELEVANT		
Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	EP 1 288 604 A (BEHR GMBH & CO [DE]) 5 March 2003 (2003-03-05) column 6, paragraph 26 - column 7, paragraph 30; figures 1-4 column 8, paragraph 33; figure 7 column 9, paragraph 41 - paragraph 42; figures 15,16	1-10
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☒ Further documents are listed in the continuation of Box C. ☒ See patent family annex.		
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Date of the actual completion of the international search 6 April 2009		Date of mailing of the international search report 15/04/2009
Name and mailing address of the ISA/ European Patent Office, P.B. 5818 Patentlaan 2 NL - 2280 HV Rijswijk Tel. (+31-70) 340-2040, Fax: (+31-70) 340-3016		Authorized officer Van Zoest, Peter

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