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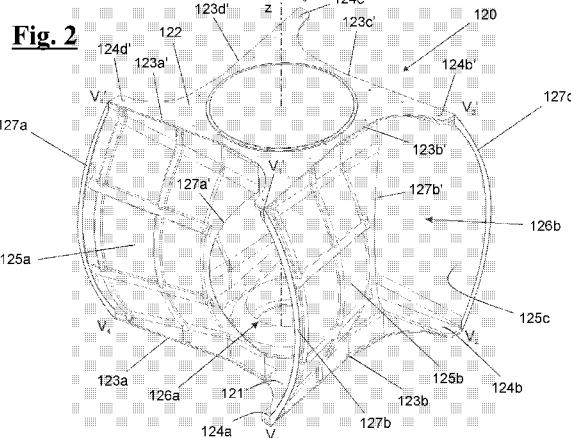
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(57) Abstract: A vertical axis wind turbine (100) comprising a rotor (110) having a substantially vertical rotation axis z, said rotor (110) comprising a plurality of blades (111) arranged to rotate about its rotation axis z in consequence of the aerodynamic drag opposed to an air flow investing said blades (111). The wind turbine (100) also comprises a casing structure (120) in which the rotor (110) houses. The casing structure (120) comprises a lower plate (121) substantially orthogonal to the rotation axis z, said lower plate (121) defined by a first plurality of vertices  $V_1, \dots, V_n$ , an upper plate (122) substantially orthogonal to the rotation axis z, said upper plate (122) defined by a second plurality of vertices  $V_1, \dots, V_n$ , each vertex  $V_1, \dots, V_n$  of the second plurality corresponding to a vertex  $V_1, \dots, V_n$  of the first plurality. The casing structure (120) also comprises a plurality of walls (123a-123d) arranged to connect the lower plate (121) and the upper plate (122), said walls (123a-123d) and said plates (121, 122) defining a plurality of openings (124a-124d) arranged to work as inlet/outlet of said air flow into/from said casing structure (120). Each wall (123a-123d) is defined by one edge (127a-127d) that connects a vertex  $V_1, \dots, V_n$  of the first plurality with a corresponding vertex  $V_1, \dots, V_n$  of the second plurality. The edge (127a-127d) is protruding outwards from the casing structure (120) with respect to a plurality of vertical ideal lines  $V_1-V_1, \dots, V_n-V_n$  joining the vertex  $V_1, \dots, V_n$  of the first plurality with the corresponding vertex  $V_1, \dots, V_n$  of the second plurality.

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TITLE

VERTICAL AXIS WIND TURBINE

DESCRIPTION

Field of the invention

5 The present invention relates to the field of wind turbines.

In particular, the invention relates to the field of vertical axis wind turbines with outer casing structure, or case.

10 Description of the prior art

Wind turbines can be subdivided into two large categories depending on the rotor direction: horizontal axis turbines and vertical axis turbines.

An example of horizontal axis turbines are the 15 windmills, where the rotation axis of the rotor is arranged substantially aligned with the prevailing air flow direction. Usually, the rotation axis can rotate in turn, usually about a vertical axis, in order to achieve and/or keep the alignment of the rotation axis of the rotor with 20 the prevailing air flow direction of. In this type of machines all the blades are invested at the same time by air flow and all the blades take part positively to the rotation of the rotor.

Horizontal axis turbines have the drawbacks of 25 requiring much space for installation and of being

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characterized by a high mechanical complexity. These drawbacks cause a high production and installation costs, besides having a big visual impact on the landscape, once installed.

5       Vertical axis turbines have a rotation axis arranged substantially orthogonal to the prevailing air flow direction. In this type of machine, an air flow doesn't produce the same effect when it hits the blades: in particular, considering the path of a single blade, it is  
10 possible to distinguish a "motor path", where air flow that acts on the blade contributes positively to the rotation of the rotor, and a "resistant path", where air flow that acts on the blade contributes negatively to the rotation of the rotor or, in any case, the contribution is less than the  
15 contribution given by "motor path".

Vertical axis turbines are characterized by a reduced mechanical complexity (with respect to horizontal axis turbines) because, for example, they are not necessarily devices that perform and/or maintain the alignment between  
20 the rotor axis and the prevailing air flow direction. Such feature explains why this type of machines has both a production cost and a maintenance cost that lower than horizontal axis machines.

However, vertical axis turbines have the drawback of  
25 having an efficiency lower than horizontal axis turbines.

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To overcome this drawback, and to increase the efficiency, different solutions have been adopted.

A first solution, provided by documents like US5852331A, JP2005299621A and WO2011/086406A2, is that of 5 shielding the "resistant path" to avoid that the air flow can act on the blade, reducing the overall efficiency.

Another possible solution, disclosed for example in WO81/03683, US5852331A, JP2005299621A and WO2011/086406A2, is that of prearranging a convergent duct that accelerates 10 air flow affecting the blades, increasing the speed of the rotor.

A further solution, described for example in FR2541383A1, consists of improving the feeding efficiency of the rotor, by directing the air flow towards the outer 15 diameter of the rotor.

However, all the wind turbines described in the above cited documents require a high air flow speed to be activated, not allowing receiving the momentum supplied by the wind in not very windy regions or in any case in 20 periods, which can be long, in which the wind doesn't reach a predetermined speed.

Furthermore, some of the above described turbines, like WO81/03683, require necessarily to be oriented with respect to the prevailing air flow direction, preventing

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exploitation of a large part of the windy components that act on the blades.

A possible solution to the problem of directing the turbine in the air flow direction is disclosed, for 5 example, in DE29980074U1 and US2011/291421A1, in which an outer case enclosing rotor and blades is provided. The outer case is furthermore equipped with a plurality of openings that operate both as inlet and outlet of the flow, according to the wind direction, and that convey air flow 10 to the blades.

However, also such solutions are not much aerodynamically efficient, since the side edge of the openings is right and does not allow neither a good flow rate nor a high flow speed as inlet.

15 Summary of the invention

It is therefore a feature of the present invention to provide a vertical axis wind turbine that has higher efficiency compared to vertical axis wind turbines of prior art.

20 It is also a feature of the present invention to provide a vertical axis wind turbine that can start the rotation of the blades at air flow speeds lower than the wind turbines of prior art.

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It is also a feature of the present invention to provide a vertical axis wind turbine that has an efficiency substantially constant responsive to the wind direction.

It is a further feature of the present invention to 5 provide a vertical axis wind turbine that has higher structural resistance both static and to fatigue with respect to the wind turbines of prior art.

It is still a feature of the present invention to provide a vertical axis wind turbine that has a production 10 and installation cost reduced with respect to the wind turbines of prior art.

It is also a feature of the present invention to provide a system of wind energy generation for arranging a plurality of wind turbines, in order to reduce, under a 15 same power installed, the space occupied with respect to systems of wind energy generation of prior art.

It is then a feature of the present invention to provide a vertical axis wind turbine that, in case of break of a component of the rotor, does not cause eject in the 20 environment potentially dangerous fragments.

These and other objects are achieved by a vertical axis wind turbine comprising:

- a rotor having a substantially vertical rotation axis z, said rotor comprising a plurality of blades 25 arranged to rotate about said rotation axis z in

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consequence of an aerodynamic drag opposed to an air flow investing said blades;

- a casing structure in which the rotor is housed.

Advantageously, the casing structure comprises:

5 - a lower plate, substantially orthogonal to the rotation axis z and defined by a first plurality of vertices  $V_1, \dots, V_n$ ;

10 - an upper plate, substantially orthogonal to the rotation axis z, and defined by a second plurality of vertices  $V'_1, \dots, V'_n$ , each vertex  $V'_1, \dots, V'_n$  of the second plurality corresponding to a vertex  $V_1, \dots, V_n$  of the first plurality;

15 - a plurality of walls arranged to connect the lower plate and the upper plate, said walls and said plates defining a plurality of openings through which an air flow is arranged to enter/exit the casing structure.

In particular, each wall is defined by a respective edge arranged to connect a vertex  $V_1, \dots, V_n$  of the first 20 plurality with a corresponding vertex  $V'_1, \dots, V'_n$  of the second plurality.

Advantageously, each edge has a profile protruding outwards from the casing structure, said protruding profile arranged to protrude beyond a corresponding straight line 25  $V_1-V'_1, \dots, V_n-V'_n$  connecting the vertex  $V_1, \dots, V_n$  of the first

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plurality with the corresponding vertex  $V_1', \dots, V_n'$  of the second plurality.

This way, an entering air flow, which enters the casing structure through a first opening, can slide on the 5 adjacent wall, which works as a convergent duct. The edge protruding outwards from such wall increases the resistance opposed to the air flow and deviates the air flow that does not enter the casing structure, so that, for most wind directions, this air flow is detached from the above 10 described wall, creating, at a second opening, adjacent to the wall opposite to the first opening, which works as outlet for an exiting air flow, a low-pressure zone that increases the flow rate of the entering air flow that enters casing structure, increasing then the wind power 15 exploited by the turbine. Furthermore, the low-pressure at second opening allows the turbine, when starting, to operate for speeds of entering air flow lower than in turbines of prior art.

Essentially, therefore, low-pressure at second 20 opening, generated by the presence of protruding edge, allows both to start more easily the turbine when the air flow is in transient state, both to increase the power when air flow is in steady state.

In particular, the edge protruding outwards has 25 curvilinear geometry with concavity oriented towards a

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respective opening, in order to act as the opening of a duct arranged to direct the air flow entering the casing structure.

More in detail, the protruding edge has a shape of 5 circular arc of predetermined radius.

Advantageously, at least one wall of the plurality of walls has a protruding portion, which starts from said edge, and configured in such a way that its projection in the plane of the lower plate is external to an ideal 10 polygon joining the vertices  $V_1, \dots, V_n$  of the lower plate to one another. The wall also comprises also a recessed portion whose projection in the plane of the lower plate is internal to the polygon, said recessed portion extending up to an opening adjacent to the wall, beyond an end of the 15 wall opposite to the edge. This way, a separation of air flow by the wall is carried out substantially for any air flow direction, and in particular also for directions parallel to the wall.

In particular, to further assist the separation of 20 the air flow also for directions of the flow parallel to the wall, the edge of the wall has a thickness higher than the remaining part of the wall.

Advantageously, the lower plate is defined by profiles that connect two by two the vertices  $V_1, \dots, V_n$  and 25 that are recessed towards the rotation axis z with respect

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to said polygon, in order to adapt to recessed portions of the walls.

Advantageously, the upper plate is defined by profiles that connect two by two the vertices  $V_1', \dots, V_n'$  and 5 that are recessed towards the rotation axis z with respect to said polygon, in order to adapt to recessed portions of the walls. This way, the air flow that slides outside on the walls does not meet the surface of the plates that would interfere with the motion of the flow, and would also 10 increase friction drag and loose energy.

In particular, the profile that defines the two plates has a point of maximum M, i.e. a point where there is the greatest distance between the profile and the corresponding edge of the polygon which connects the 15 vertices  $V_1, \dots, V_n$  or the vertices  $V_1', \dots, V_n'$ . This point of maximum M is located near the above described end of the walls. Each wall then maintains a substantially constant inclination from the edge to the end, beyond which the wall breaks, thus leaving space to the adjacent opening. This 20 way, each wall, in addition to work as obstacle for creating the low-pressure at the outlet, works also as convergent duct for the air flow that enters the casing structure.

Advantageously, each profile that joins two plates 25 has a tangent line between  $0^\circ$  and  $30^\circ$  with respect to a

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corresponding edge of an ideal polygon that connects the vertices  $V_1, \dots, V_n$ , to the farthest vertex from recessed portion of the wall. This way, the profile meets the inclination of the wall, which works as convergent duct

5 with inclination of  $30^\circ$ .

Advantageously, each profile joining the plates comprises, at the corresponding opening, a straight portion which is used to strengthen the structure, giving to the wall a stiffening effect that prevents it from bending when

10 it is subjected to heavy aerodynamic loads.

Advantageously, on the outer surface of the walls ribs with substantially reticular geometry are provided, that, in addition to assist the structural resistance of the walls, further assist the separation of the air flow,

15 and, in particular of the boundary layer, from the walls.

In particular, the casing structure comprises four walls, which are of identical shape and are at an angle of  $90^\circ$  from each other about axis  $z$ , which define four openings configured to work as inlet/outlet for the air

20 flow in/from the casing structure. Such configuration is aerodynamically optimal and allows an efficiency that is much less variable responsive to changes of the wind direction. Furthermore, by having identical components, it reduces the production costs and assists the assembly of

25 the turbine.

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Advantageously, a first opening among the four openings is configured to allow a main air flow component to enter, said first opening being adjacent to a first wall of the four walls and to a second wall of the four walls, 5 substantially orthogonal to the first wall. This way, when the main air flow component enters first opening, part of the air flow is hampered by the first and second walls, creating a low pressure zone at a second opening, adjacent to the second wall, and at a fourth opening, adjacent to 10 the first wall, a third opening being opposite to the first opening.

This way, the low-pressure zone at the second opening allows both to start more easily the turbine when the air flow is in transient state, both to increase the power when 15 air flow is in steady state. At the same time, the low-pressure zone at the fourth opening contributes to cause the air flow to exit from the casing structure, through the fourth opening, allowing emptying the "resistant path" of the rotor and thus reducing the resistance encountered by 20 the rotor in its rotation. Such lower resistance causes an improvement of the overall efficiency of the wind turbine, allowing to further lower the value of the wind speed for which the turbine comes into operation at start up.

According to another aspect of the invention, a 25 system of wind energy generation comprises at least two

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wind turbines, according to the present invention, which are located above each other, in such a way that the rotation axes z of the wind turbines are coincident. This way, it is possible to create systems of wind power generation that require a surface of installation substantially the same as the surface necessary to a single turbine. This allows to strongly optimize the space with respect to the systems of wind power generation of prior art, which usually provide the arrangement of several wind turbines in a triangle or in a square, thus needing a surface of installation much bigger than that required for a single turbine. Furthermore, owing to the nature of the wind turbine according to the invention, such an arrangement in column does not affect the performance of the individual turbines with respect to each other.

Advantageously, the wind turbines are rotationally shifted with respect to the rotation axis z of a predetermined angle from each other.

In particular, the predetermined angular offset is set between  $20^\circ$  and  $40^\circ$ , advantageously, it is equal to  $30^\circ$ .

This way, it is further increased, with respect to the single wind turbine, the invariance of the performances of the wind system responsive to the wind direction. Furthermore, this arrangement with angular offset

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distributes in a more balanced way the aerodynamic loads, especially fatigue loads that would affect the system, reducing the possibility of resonance phenomena and allowing a higher life of the components.

5           Brief description of the drawings

Further characteristic and/or advantages of the present invention will be made clearer with the following description of some exemplary embodiments, exemplifying but not limitative, with reference to the attached drawings in  
10 which:

- Fig. 1 shows, in a first exemplary embodiment, the rotor of a vertical axis wind turbine, according to the present invention;
- Fig. 2 shows, in a first exemplary embodiment, the casing structure of a vertical axis wind turbine, according to the present invention;
- Fig. 3 shows a vertical axis wind turbine comprising the rotor and the casing structure of Figs. 1 and 2;
- Fig. 4 shows, in a horizontal plane, a vertical axis wind turbine, according to the present invention, crossed by an air flow;
- Fig. 5 shows, in a horizontal plane and in detail, the vertical axis wind turbine of Fig. 4;
- Fig. 6 shows a system of wind energy generation,

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according to the present invention, in a first exemplary embodiment;

- Fig. 7 shows a system of wind energy generation, according to the present invention, in a second exemplary embodiment.

5

#### Description of a preferred exemplary embodiment

With reference to Figs. 1, 2 and 3, an exemplary embodiment of a vertical axis wind turbine 100, according to the present invention, comprises a rotor 110 (Fig.1) and 10 a casing structure 120 (Fig.2) that houses the rotor 110.

With reference to Fig. 1, rotor 110 comprises a plurality of blades 111, in particular 3, which are arranged in an axial-symmetric way with respect to a vertical rotation axis z, about which the blades 111 are 15 wheeled, as consequence of the aerodynamic drag opposed to an air flow that invest blades 111.

With reference to Fig. 2, casing structure 120 comprises a lower plate 121 and an upper plate 122, substantially orthogonal to the axis z, and defined, 20 respectively, by a first plurality of vertices, in particular four vertices,  $V_1$ ,  $V_2$ ,  $V_3$ ,  $V_4$  and a second plurality of vertices, in particular four vertices,  $V'_1$ ,  $V'_2$ ,  $V'_3$ ,  $V'_4$ . The two plates 121 and 122 are connected to each other through a plurality of walls, in particular four 25 walls, 125a-125d, which create, with the two plates

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121, 122, four openings 126a-126d that work, according to  
the direction of air flow that crosses the casing structure  
120, as air flow inlet and/or outlet ports. The walls 125a-  
125d, as well as all the other side elements of the casing  
5 structure 120, are equal to each other and repeated for  
each one of four sides V<sub>1</sub>-V<sub>2</sub>, V<sub>2</sub>-V<sub>3</sub>, V<sub>3</sub>-V<sub>4</sub>, V<sub>4</sub>-V<sub>1</sub>.

Such exemplary embodiment provides that each of walls  
125a-125d is defined by edges 127a-127d, each of which  
connects one of the vertices V<sub>1</sub>, V<sub>2</sub>, V<sub>3</sub>, V<sub>4</sub> with one of the  
10 corresponding vertices V<sub>1</sub>', V<sub>2</sub>', V<sub>3</sub>', V<sub>4</sub>'. In particular,  
each edge 127a-127d is protruding outwards from the casing  
structure 120 with respect to the corresponding vertical  
line that connects one of the vertices V<sub>1</sub>, V<sub>2</sub>, V<sub>3</sub>, V<sub>4</sub> with  
one of the corresponding vertices V<sub>1</sub>', V<sub>2</sub>', V<sub>3</sub>', V<sub>4</sub>'.

15 This way, with reference to Fig. 4, an air flow that  
enters casing structure 120 through, for example, a first  
opening 126a, slides on adjacent wall 125b, which works as  
convergent duct. The edge 127b, protruding outwards from  
wall 125b, increases the resistance opposed to the air flow  
20 and deviates the air flow that does not enter casing  
structure 120, so that, for the majority of the wind  
directions, this air flow is separated by the wall 125b  
creating, at a second opening 126b, which is adjacent to  
the wall 125b and that works as outlet of the air flow, a  
25 low-pressure zone that increases the air flow rate that

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enters casing structure 120, increasing then the wind power exploited by the turbine 100. Furthermore, the low-pressure at the second opening 126b allows the turbine to operate for start up speeds of inlet air flow lower than turbines 5 of prior art.

In essence, therefore, the low-pressure at opening 126b, generated by the presence of protruding edge 127b, allows both to assist the start up of turbine 100 when the air flow is in transient state, and to increase the power 10 when the air flow is in steady state.

Furthermore, wall 125a, thanks to edge 127a, which is also protruding, opposes a resistance against the air flow that allows an immediate separation of such air flow, generating, in a similar way as above said, a low-pressure 15 zone at a fourth opening 126d, which is also useful for the exit of the air flow from the casing structure 120.

Such low-pressure zone on the fourth opening 126d contributes to cause the air flow to exit from the casing structure 120, through the opening 126d, allowing emptying 20 the "resistant path" of the rotor 110 and then reducing the resistance encountered by the rotor 110 in its rotation. A lower resistance causes an improvement of the overall efficiency of the wind turbine 100, allowing further to lower the value of the wind speed for turbine 100 at the 25 start up.

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In particular, the protruding edge 127b has a curvilinear shape, as a circular arc with concavity oriented towards the opening 126a, which allows the opening 126a to better convey the air flow entering the casing 5 structure 120, assuming a shape similar to that of an inlet duct.

Furthermore, with reference to Fig. 5, considering an ideal square (shown as a dotted line in the figures) that connects the vertices  $V_1'$ ,  $V_2'$ ,  $V_3'$ ,  $V_4'$  of the upper plate 10 122, or similarly the vertices  $V_1$ ,  $V_2$ ,  $V_3$ ,  $V_4$  of the lower plate 121, the walls 125a-125d comprise a protruding portion 125a'-125d', configured in such a way that its projection in the plane of the lower plate 121 is external to the square, and a recessed portion 125a''-125d'', whose 15 projection in this plane instead remains inside the square itself.

With reference again, for example, to the flow passing through the opening 126a, the protruding portion 125b' comprises edge 127b and, as said, is used for 20 increasing the resistance opposed to the air flow, in order to allow the separation from wall 125b. The recessed portion 125b'', instead, causes the low-pressure generated by said flow separation to occur exactly at outlet 126b. In particular, this embodiment allows that the phenomenon 25 above described occurs substantially for any direction of

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the wind, even for those parallel to a wall 125a-125d, which is the more critic direction, at which it is much more difficult the separation of the air flow. To further assist the separation in this direction, in addition, also 5 the edge 127a-127d may have a thickness greater than the rest of the wall 125a-125d, as shown in Figs. 2 and 3.

Still with reference to Figs. 2 and 3, the plates 121 and 122 are joined by profiles 123a-123d and 123a'-123d' that connect two by two the vertices  $V_1$ ,  $V_2$ ,  $V_3$ ,  $V_4$  and the 10 vertices  $V_1'$ ,  $V_2'$ ,  $V_3'$ ,  $V_4'$ , respectively. Such profiles 123a-123d and 123a'-123d' are curvilinear and are recessing towards the rotation axis z, in order to meet the shape of the walls 125a-125d, and, in particular of recessing portions 125a''-125d''. This way, an air flow that slides 15 outside on walls 125a-125d does not encounter the surface of the plates that would interfere with the motion of the flow, and would also increase the friction drag and loose energy.

Geometrically, then, profiles 123a-123d and 123a'-123d' substantially follow the trend of walls 125a-125d. 20

With reference to Figs. 2, 3 and 5, in an exemplary embodiment, profiles 123a-123d and 123a'-123d' have a point of maximum M, i.e. a point at which the distance between the profile and the side of said square is greatest. Such 25 point of maximum M is located close to the recessed portion

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125a''-125d'', and, in particular at one end 127a'-127d' of the wall 125a-125d opposite to the edge 127a-127d. This way, the wall 125b, and similarly the other walls 125a, 125c, 125d, maintains an inclination substantially constant 5 from edge 127a up to second edge 127a', beyond which the wall 125b is interrupted leaving space to the adjacent opening 126b.

This way, considering Fig. 4, the wall 125b, in addition to act as an obstacle to create the low-pressure 10 at outlet 126b, acts also as convergent duct for the air flow that enters the casing structure. In particular, with reference to Fig. 5, the wall 125b, and similarly the other walls, has an inclination of about 30° with respect to the segment V<sub>1</sub>'-V<sub>2</sub>', or to the segment V<sub>1</sub>-V<sub>2</sub>. The profiles 123b 15 and 123b', which must follow the wall 125b, have a tangent line at vertices V<sub>1</sub> and V<sub>1</sub>' that is about 30°. The tangent line of profiles 123b and 123b' at the vertices V<sub>2</sub> and V<sub>2</sub>', instead, is about 30° with respect to the segments V<sub>2</sub>-V<sub>3</sub> and V<sub>2</sub>'-V<sub>3</sub>', in order to allow a wall 125c to be in turn a 20 convergent duct with respect to the inlet 126b. With reference to Fig. 2, profiles 123b and 123b' comprise, in the part adjacent to vertices V<sub>2</sub> and V<sub>2</sub>', straight portions 124b and 124b' that serve to strengthen the structure, giving to the wall 125c a stiffening effect that prevents 25 it from bending when it is subjected to large aerodynamic

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loads. For the same reason, on the outer surface of walls 125a-125d, ribs with substantially reticular geometry are provided. Such stiffening ribs, in addition to contribute to the structural strength of the walls 125a-125d, also 5 contribute to further facilitate the separation of the air flow, and, in particular of the boundary layer, from the walls.

With reference to Fig. 6, the present invention provides a system of wind energy generation 200 comprising 10 more than one wind turbine 100 as previously described. The turbines 100 are arranged above one another aligned with respect to vertical axis z. This way, systems of wind energy generation 200 can be obtained that require an installation surface substantially the same as the surface 15 necessary to a single turbine 100. This allows to strongly optimize the space with respect to the systems of wind energy generation of the prior art, which usually provide the arrangement of multiple wind turbines in a triangle or in a square, thus requiring a surface of installation much 20 bigger than a single turbine. Furthermore, for the nature of wind turbines 100, according to the invention, this arrangement in column does not affect the performances of singles turbines 100.

With reference to Fig. 7, an exemplary embodiment of 25 the system of wind energy generation 200 provides that the

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wind turbines 100 are rotationally shifted at a predetermined angle about its axis z with respect to each other, in order to further increase, with respect to the single wind turbine 100 and to the system 100 of Fig. 6, 5 the invariance of the performances of the wind system 100 respect to the direction of the wind.

Such predetermined angle is advantageously set between 20° and 40°, and in particular 30°. Anyway, the optimal value of this angle can result different depending 10 on the number of wind turbines 100 arranged in column.

Furthermore, the arrangement of the turbines 100 of Fig. 7 distributes in a more balanced way the aerodynamic loads, especially fatigue loads, which the system 100 must endure, reducing the possibility of resonance phenomena and 15 allowing a higher life of the components.

Although in the present description reference has been made with to an exemplary embodiment of the casing structure comprising four walls, also exemplary embodiments where the casing structure comprises a different number of 20 walls are comprised .

The foregoing description of specific exemplary embodiments will so fully reveal the invention according to the conceptual point of view, so that others, by applying current knowledge, will be able to modify and/or adapt in 25 various applications of the specific exemplary embodiments

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without further research and without parting from the invention, and, accordingly, it is meant that such adaptations and modifications will have to be considered as equivalent to the specific embodiments. The means and the 5 materials to realise the different functions described herein could have a different nature without, for this reason, departing from the field of the invention. it is to be understood that the phraseology or terminology that is employed herein is for the purpose of description and not 10 of limitation.

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CLAIMS

1. A vertical axis wind turbine (100) comprising:

- a rotor (110) having a substantially vertical rotation axis z, said rotor (110) comprising a plurality of blades (111) arranged to rotate about said rotation axis z in consequence of an aerodynamic drag opposed to an air flow investing said blades (111);
- a casing structure (120) in which said rotor (110) is housed, said casing structure (120) comprising:
  - a lower plate (121) substantially orthogonal to said rotation axis z, said lower plate (121) defined by a first plurality of vertices  $V_1, \dots, V_n$ ;
  - an upper plate (122) substantially orthogonal to said rotation axis z, said upper plate (122) defined by a second plurality of vertices  $V'_1, \dots, V'_n$ , each vertex  $V'_1, \dots, V'_n$  of said second plurality corresponding to a vertex  $V_1, \dots, V_n$  of said first plurality;
  - a plurality of walls (125a-125d) arranged to connect said lower plate (121) and said upper plate (122), said walls (125a-125d) and said plates (121, 122) defining a plurality of openings (126a-126d) through which said air flow

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is arranged to enter/exit said casing structure (120);

5 said wind turbine (100) characterized **in that** each wall (125a-125d) of said plurality of walls is defined by a respective edge (127a-127d) arranged to connect a vertex  $V_1, \dots, V_n$  of said first plurality with a corresponding vertex  $V_1', \dots, V_n'$  of said second plurality,

10 **and in that** each said edge (127a-127d) has a profile protruding outwards from said casing structure (120), said protruding profile arranged to protrude beyond a corresponding straight line  $V_1-V_1', \dots, V_n-V_n'$  connecting said vertex  $V_1, \dots, V_n$  of said first plurality with said corresponding vertex  $V_1', \dots, V_n'$  of said second plurality, in 15 such a way that when an entering air flow enters said casing structure (120) through a first opening (126a), said protruding profile of said edge (127b) generates the separation of a portion of flow from said wall (125b) creating, at a second opening (126b) adjacent to said wall 20 (125b) opposite to said first opening (126a), a low-pressure zone that increases the flow rate of said entering air flow that enters said casing structure (120).

2. The wind turbine (100), according to claim 1, wherein said edge (127a, 127b, 127c, 127d) has a 25 curvilinear geometry with concavity oriented towards

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a respective opening (126d, 126a, 126b, 126c).

3. The wind turbine (100), according to claim 1, wherein at least one wall (125a-125d) of said plurality of walls has a protruding portion (125a'-125of), configured in such a way that its projection in the plane of said lower plate (121) is external to an ideal polygon joining the vertices  $V_1, \dots, V_n$  of said lower plate (121) to one another, said protruding portion (125a'-125of) starting from said edge (127a-127d), and a recessed portion (125a''-125d'') whose projection on said plane of said lower plate (121) is internal to said polygon, said recessed portion (125a''-125d'') extending up to an opening (126a-126d) adjacent to said wall (125a-125d) beyond an end (127a'-127of) of said wall opposite to said edge (127a-127d), said recessed portion (125a''-125of) configured in order to assist the air flow separation from said wall (125a-125d) for each direction of said air flow.

20 4. The wind turbine (100), according to claim 3, wherein at least one among said lower plate (121) and said upper plate (122) is defined by profiles (123a-123d, 123a'-123d') connecting two by two said vertices  $V_1, \dots, V_n$  of said first plurality and/or said 25 vertices  $V_1', \dots, V_n'$  of said second plurality, said

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profiles (123a-123d, 123a'-123of) being recessed toward said rotation axis z with respect to said polygon, in such a way to fit to said recessed portions (125a''-125d'') of said walls (125a-125d) of 5 said plurality of walls.

5. The wind turbine (100), according to claim 4, wherein said profiles (123a-123d, 123a'-123of) have a point of maximum M near said end (127a'-127of) of 10 said walls (125a-125d), said point of maximum M being the point of said profiles (123a-123d, 123a'-123of) that mostly distances itself by the corresponding segment that connects two by two said vertices  $V_1, \dots, V_n$  and/or said vertices  $V_1', \dots, V_n'$ .

6. The wind turbine (100), according to any of the 15 previous claims, wherein said casing structure (120) comprises four walls (125a-125d) of identical shape located at an angle of 90° from each other about said axis z, said four walls (125a-125d), defining four openings (126a-126d), arranged to act as inlet/outlet 20 of said air flow in/from said casing structure (120).

7. The wind turbine (100), according to claim 6, wherein a first opening (126a) of said four openings (126a-126d) is configured to allow a main component of said air flow to enter, said first opening (126a) 25 being adjacent to a first wall (125a) of said four

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walls (125a-125d) and to a second wall (125b) of said four walls (125a-125d), substantially orthogonal to said first wall (125a), in such a way that, when said main component of said air flow enters said first opening (126a), part of said air flow is hampered by said first and second wall (125a, 125b), creating a low pressure zone at a second opening (126b), adjacent to said second wall (125b), and at a fourth opening (126d), adjacent to said first wall (125a), a third opening (126c) being opposite to said first opening (126a).

8. The wind turbine (100), according to any of the previous claims, wherein on the outer surface of said walls (125a-125d) ribs are provided configured to increase the structural resistance of said walls (125a-125d) and to assist a separation of the boundary layer of said air flow from said walls (125a-125d).

9. System of wind energy generation (200) comprising at least two wind turbines (100) according to any of the previous claims, said at least two wind turbines (100) being located above each other, in such a way that said rotation axes z of said wind turbines (100) are coincident.

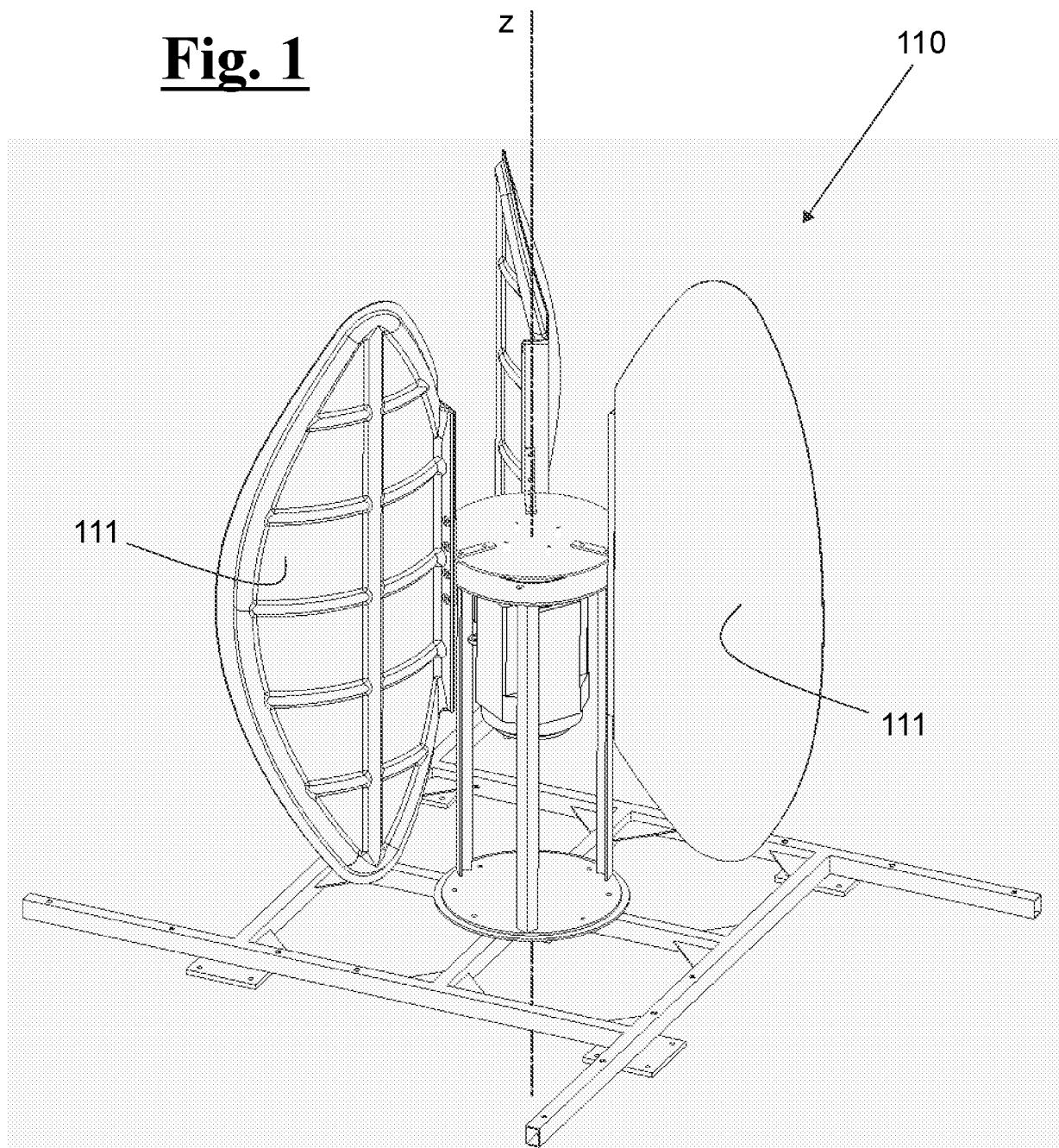
25 10. System of wind energy generation (200),

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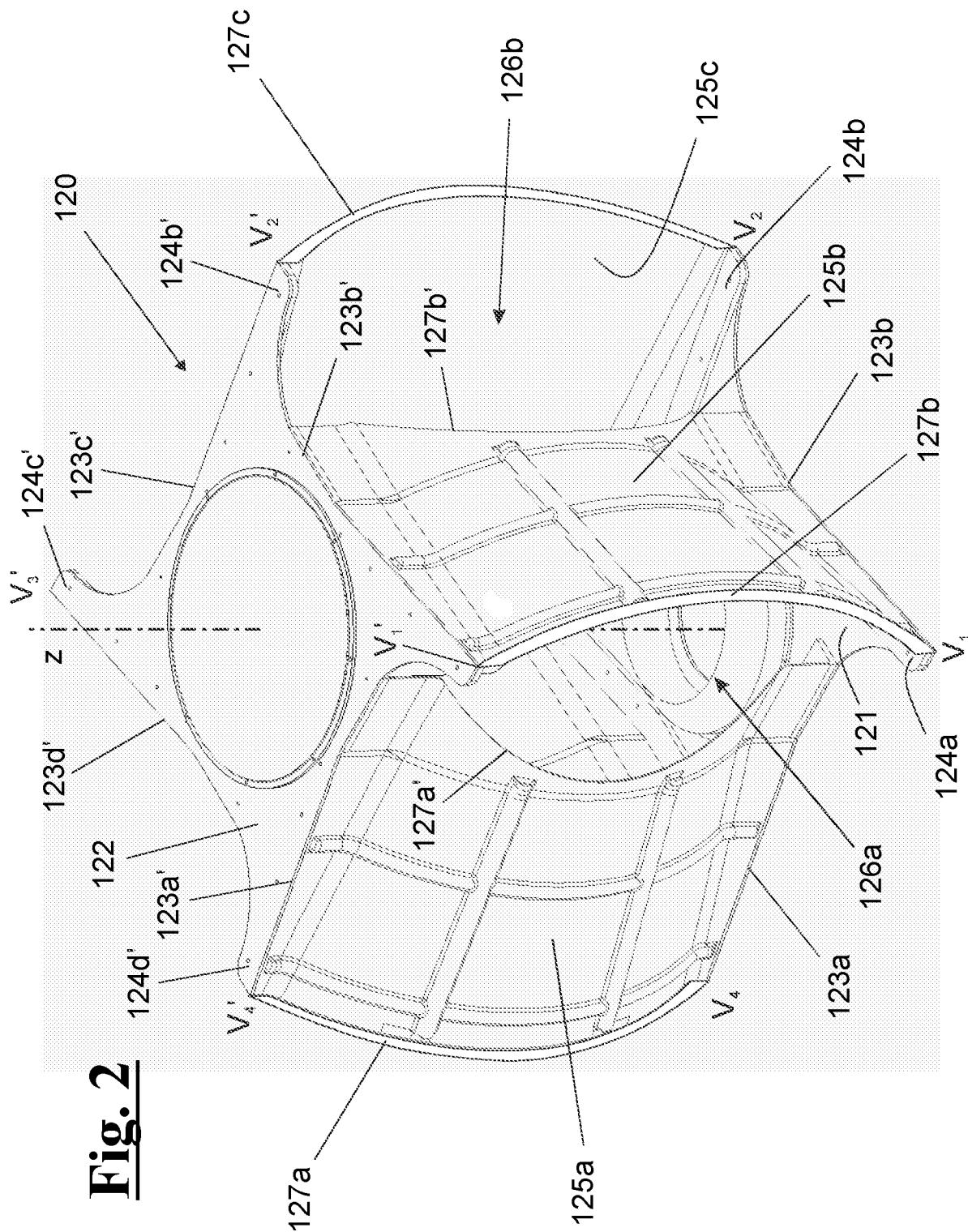
according to claim 9, wherein said wind turbines (100) are rotationally shifted with respect to said rotation axis z of a predetermined angle from each other set between 20° and 40°.

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**Fig. 1**



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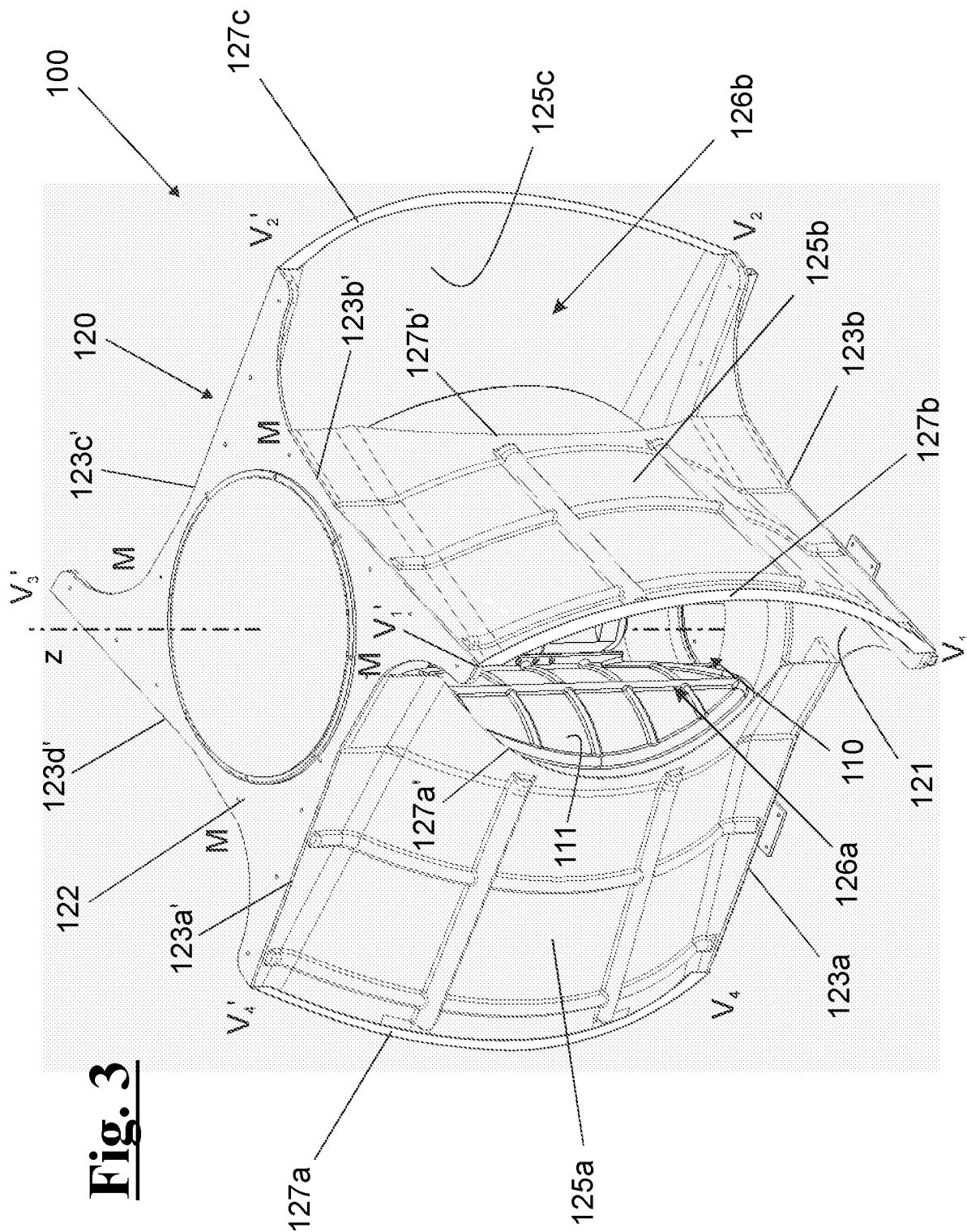
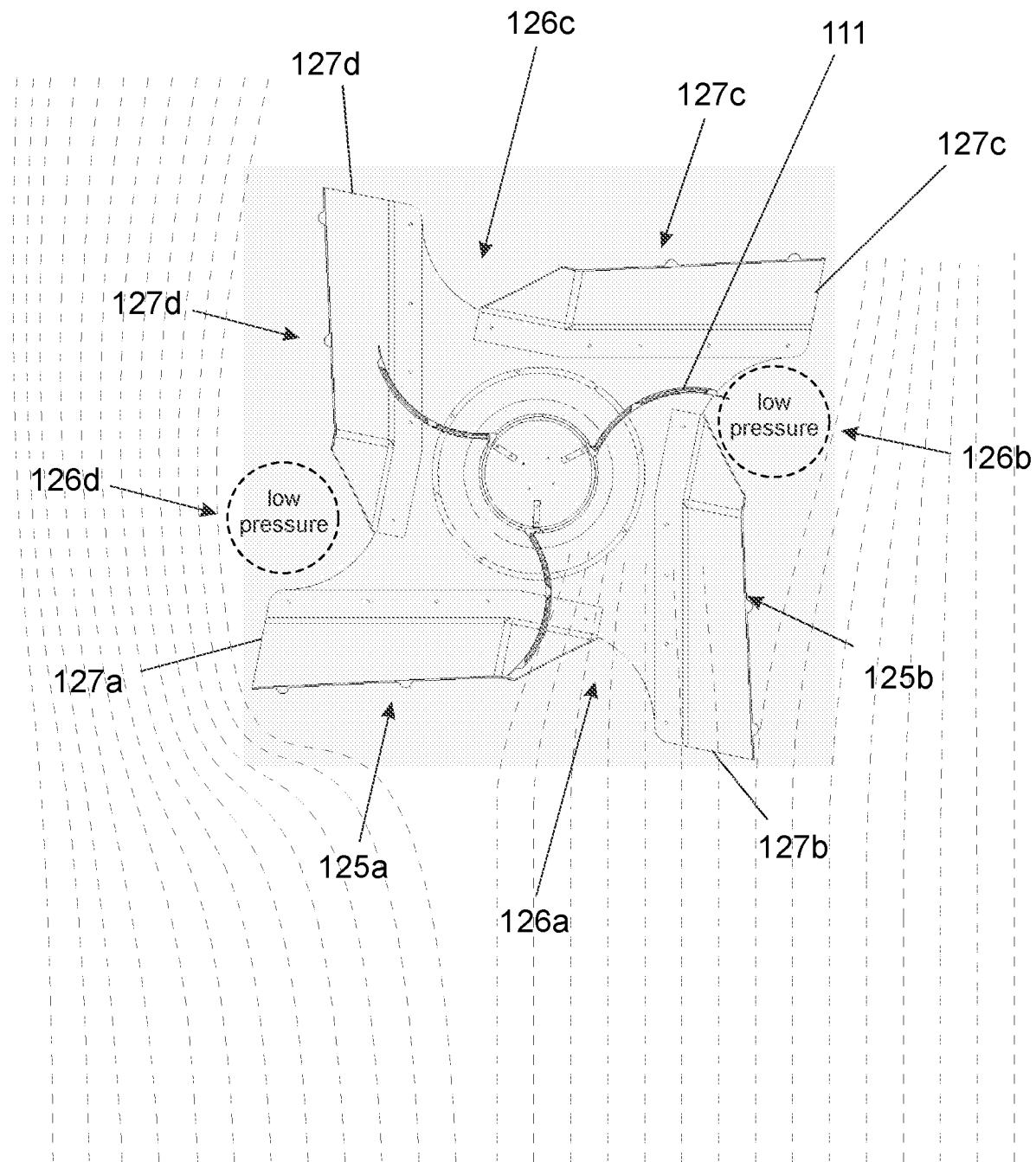


Fig 3

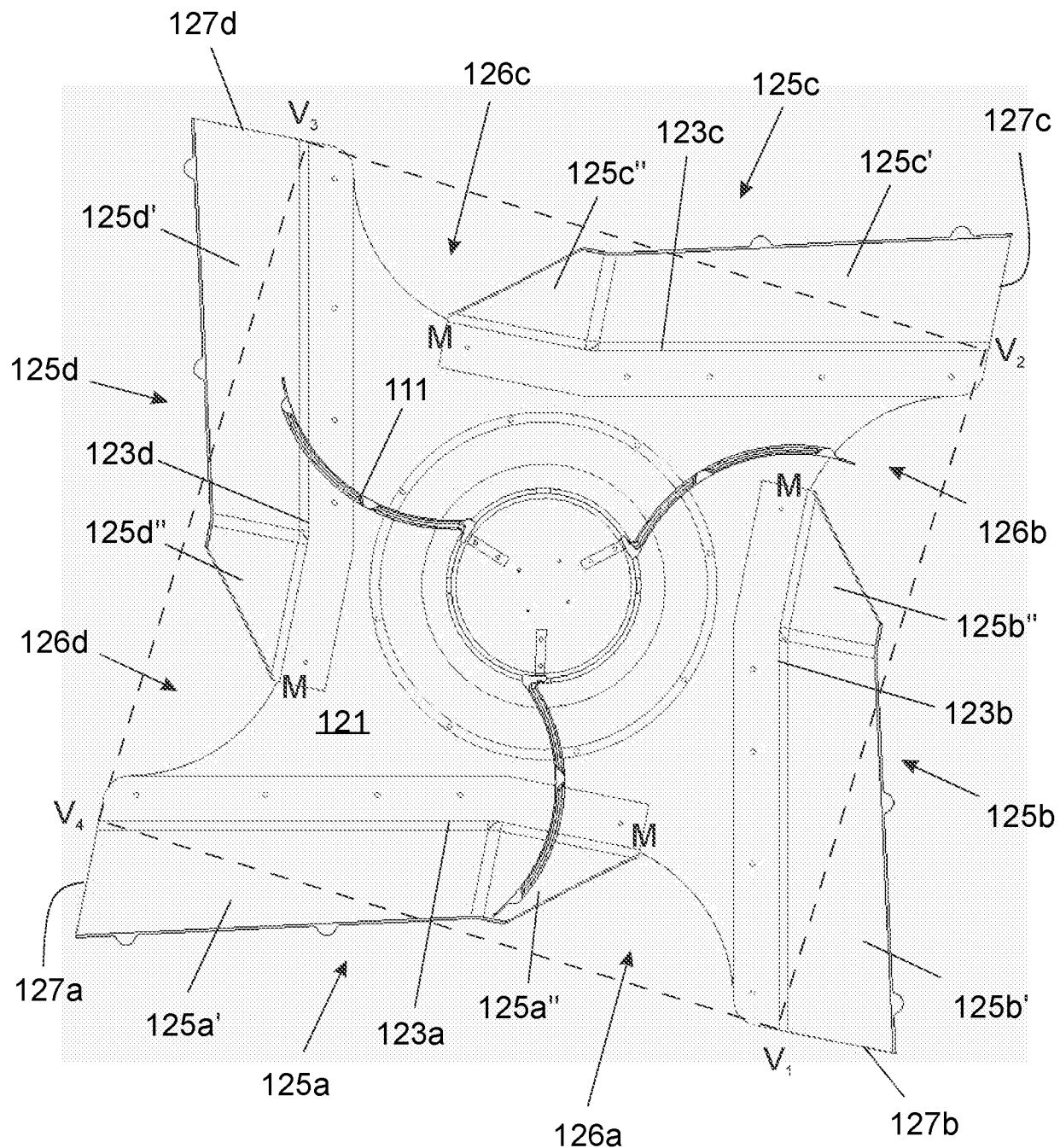
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**Fig. 4**



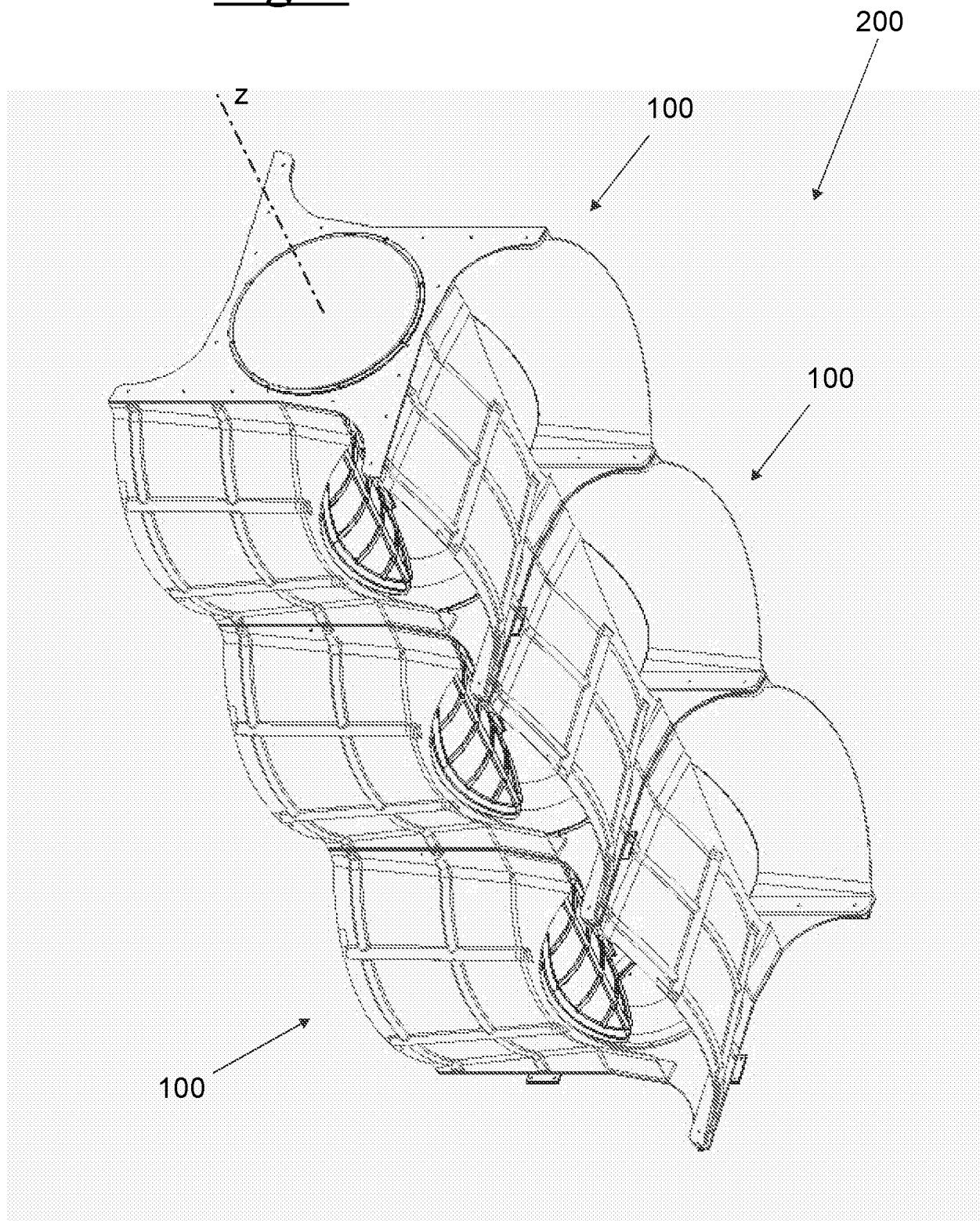
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**Fig. 5**



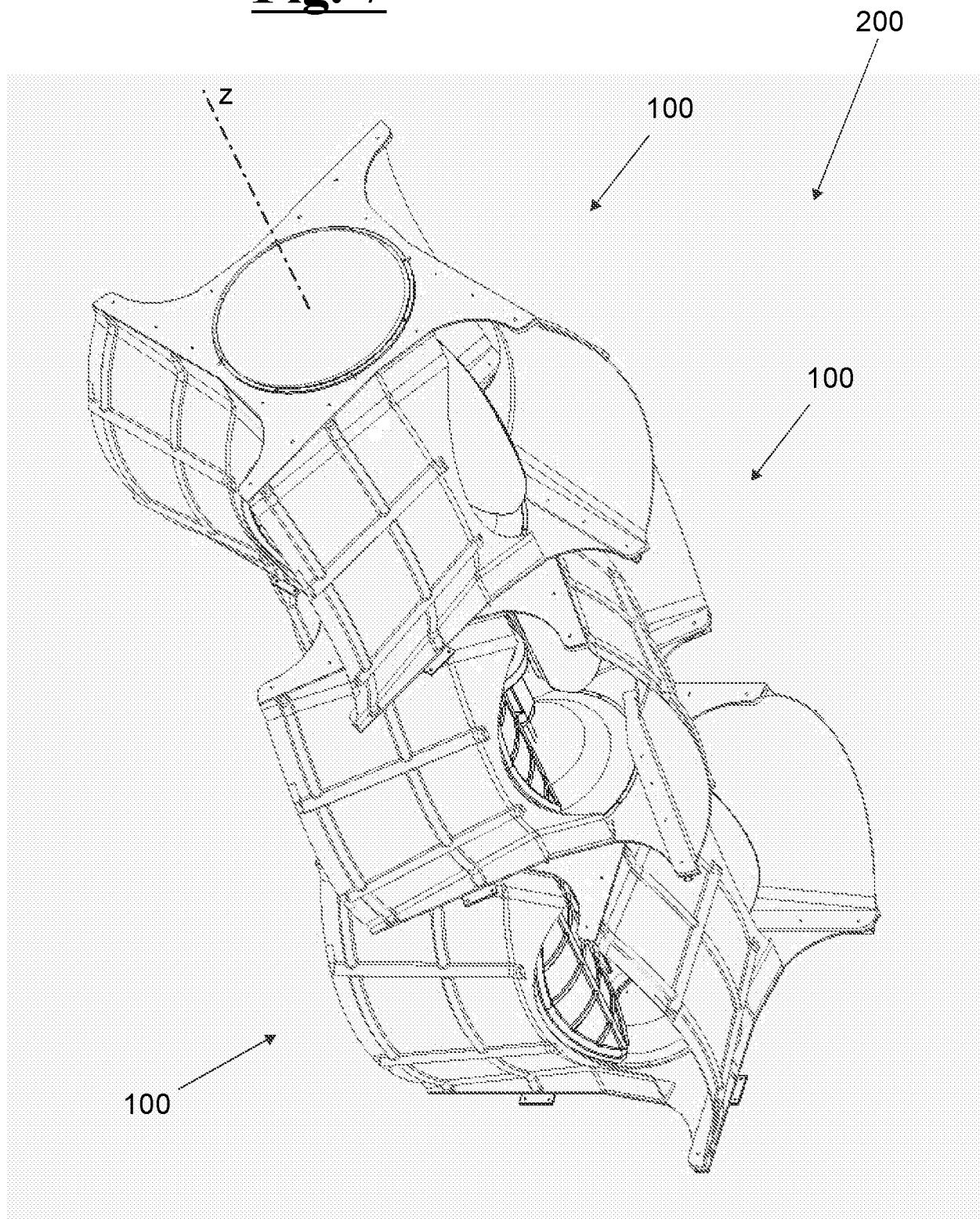
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**Fig. 6**



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**Fig. 7**



# INTERNATIONAL SEARCH REPORT

International application No  
PCT/IB2014/062913

**A. CLASSIFICATION OF SUBJECT MATTER**  
INV. F03D3/04 F03D3/02  
ADD.

According to International Patent Classification (IPC) or to both national classification and IPC

**B. FIELDS SEARCHED**

Minimum documentation searched (classification system followed by classification symbols)

F03D F03B

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)

EPO-Internal, WPI Data

**C. DOCUMENTS CONSIDERED TO BE RELEVANT**

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	WO 2011/001375 A1 (BIUCCHI GABRIELE [IT]; MANTOVANI MARCO [CH]) 6 January 2011 (2011-01-06) page 3, line 16 - page 4, line 29; figures 1,3	1-8
Y	US 2010/230974 A1 (CANTWELL LARRY [US]) 16 September 2010 (2010-09-16) abstract; figure 1	9,10
Y	-----	9,10



Further documents are listed in the continuation of Box C.



See patent family annex.

\* Special categories of cited documents :

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Date of the actual completion of the international search	Date of mailing of the international search report
21 November 2014	28/11/2014
Name and mailing address of the ISA/ European Patent Office, P.B. 5818 Patentlaan 2 NL - 2280 HV Rijswijk Tel. (+31-70) 340-2040, Fax: (+31-70) 340-3016	Authorized officer  Król, Marcin

**INTERNATIONAL SEARCH REPORT**

Information on patent family members

International application No

PCT/IB2014/062913

Patent document cited in search report	Publication date	Patent family member(s)	Publication date
WO 2011001375	A1 06-01-2011	NONE	
US 2010230974	A1 16-09-2010	NONE	