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Herbert et al.

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- (54) **FLOW NOISE RESTRICTOR**
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 See application file for complete search history.

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E03D 3/04 (2006.01)

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(58) **Field of Classification Search**
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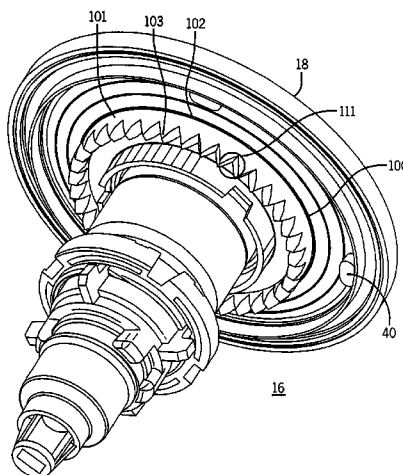
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(57) **ABSTRACT**
A flow noise restrictor for use with a valve. The flow noise restrictor reduces the flow area as the valve closes and forms vortices to reduce the noise such as due to the Bernoulli effect.

13 Claims, 12 Drawing Sheets



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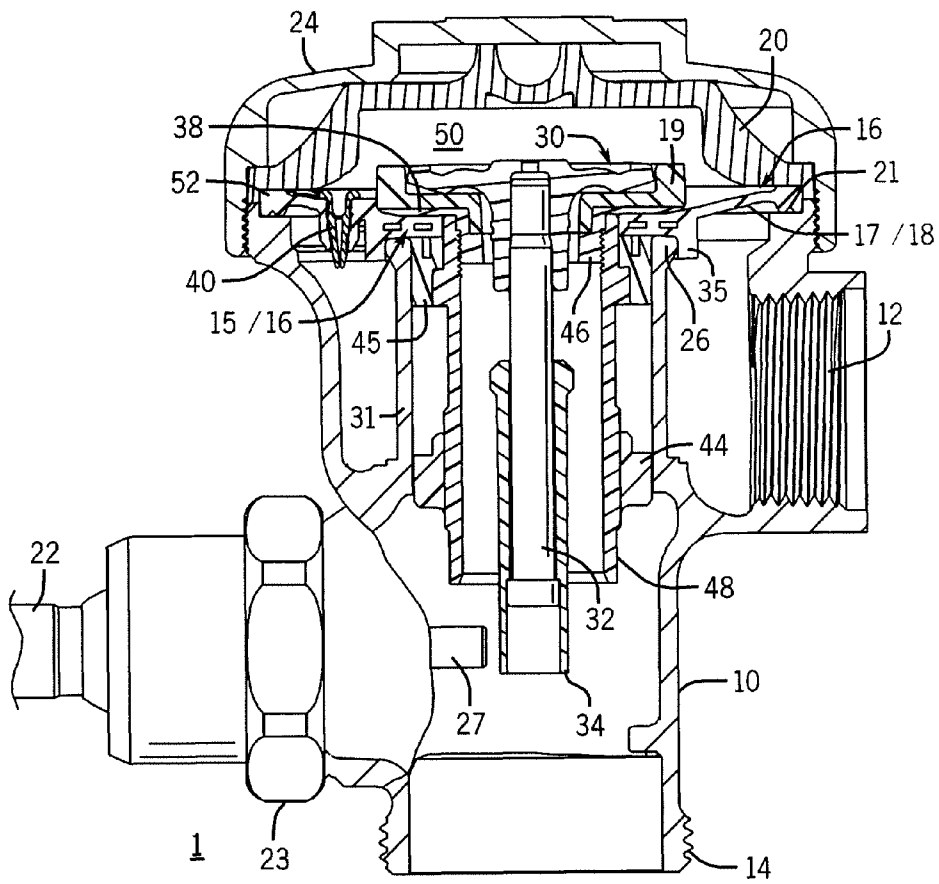
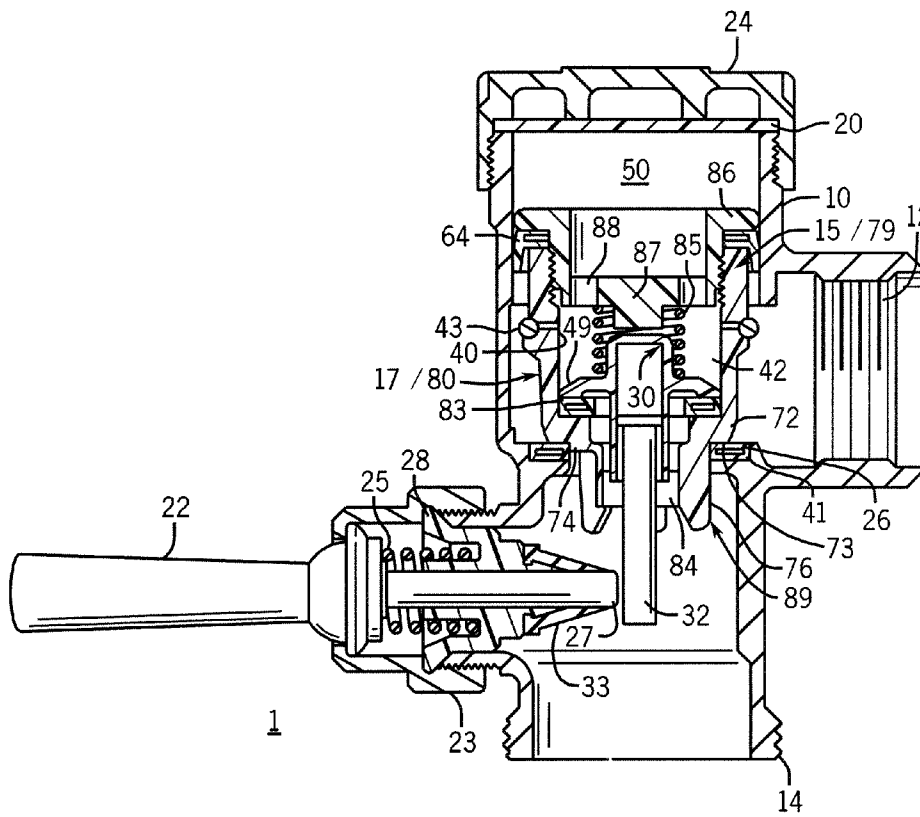
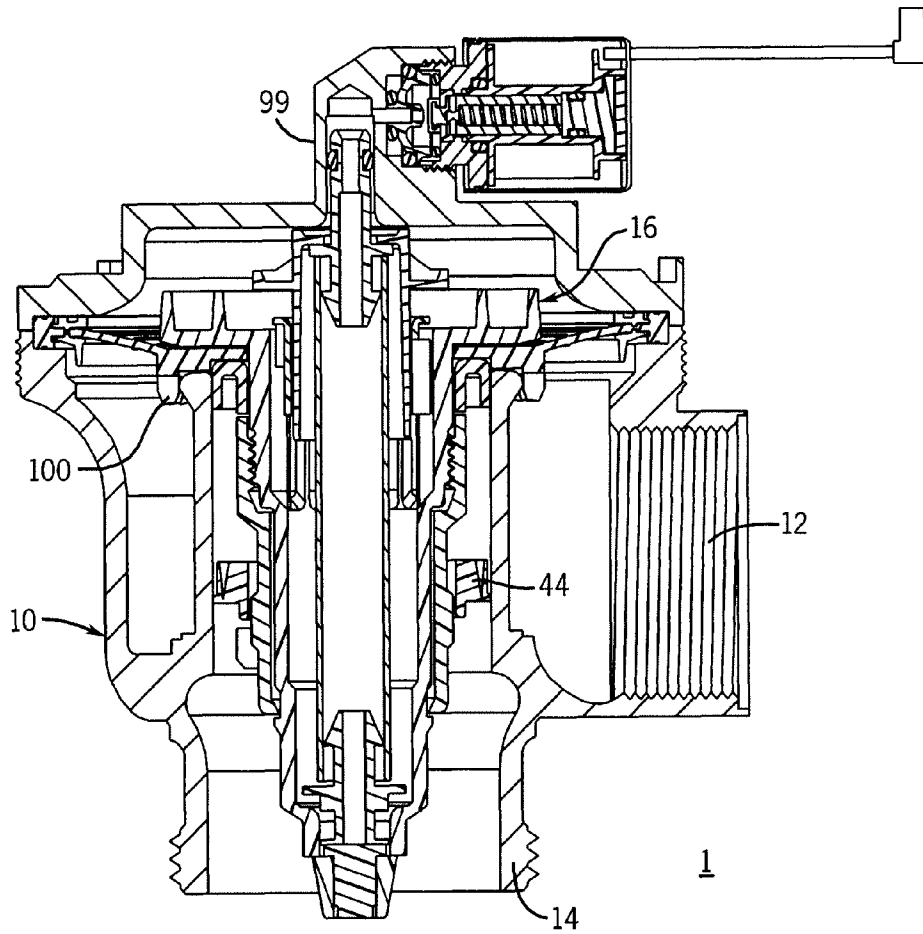


FIG. 1





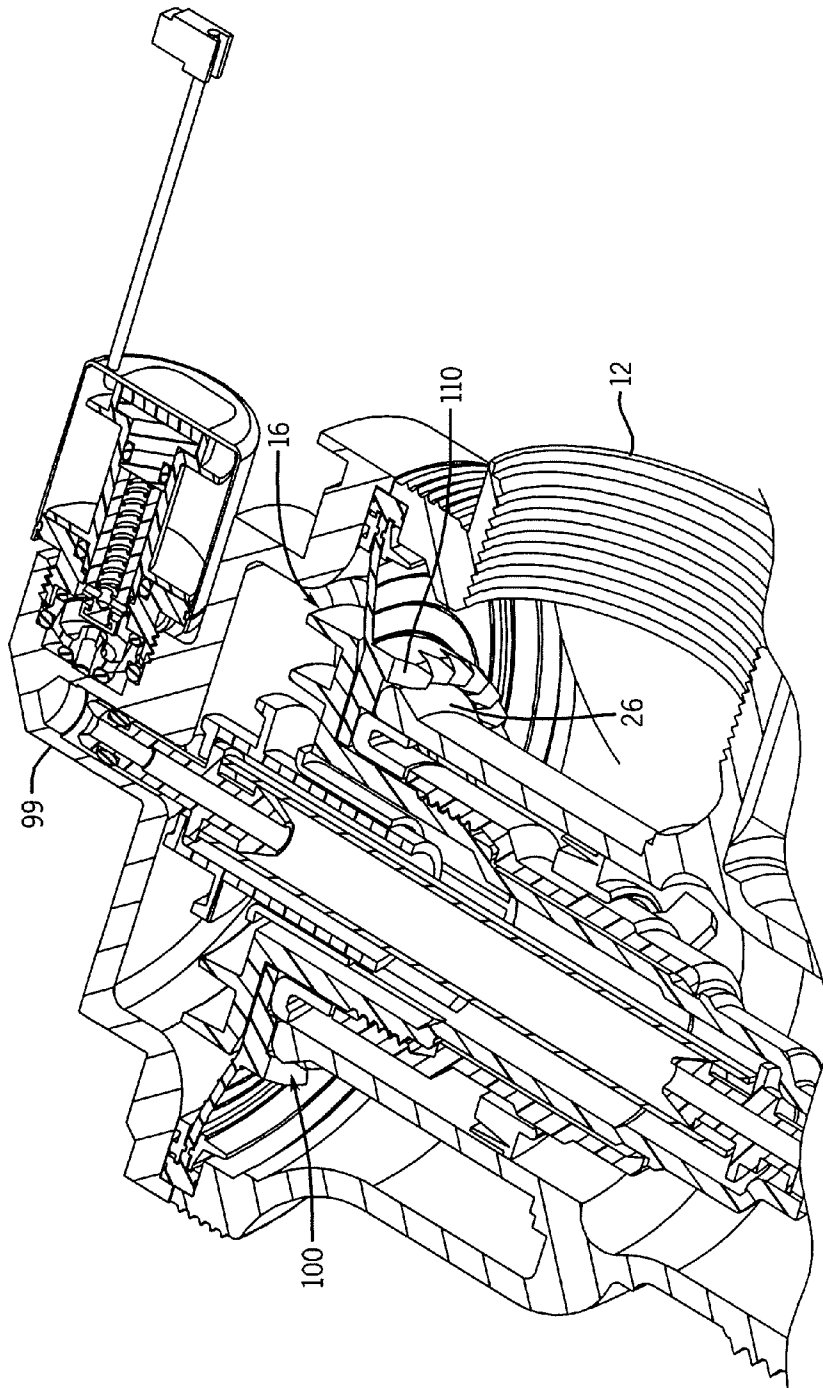


FIG. 3B

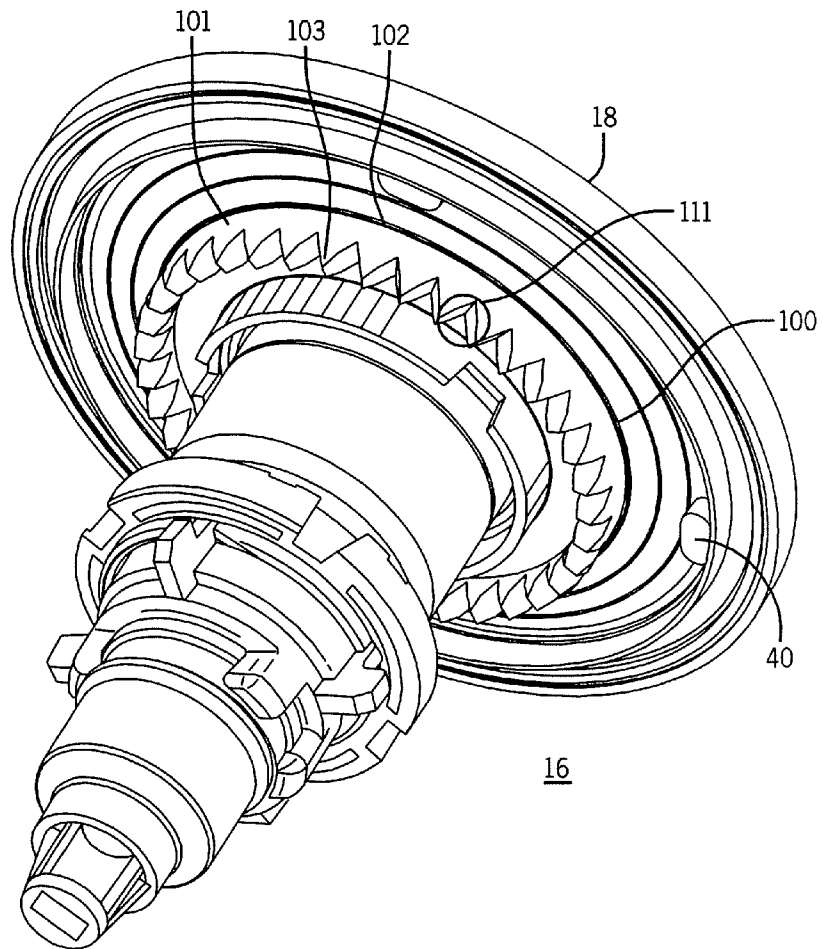


FIG. 4A

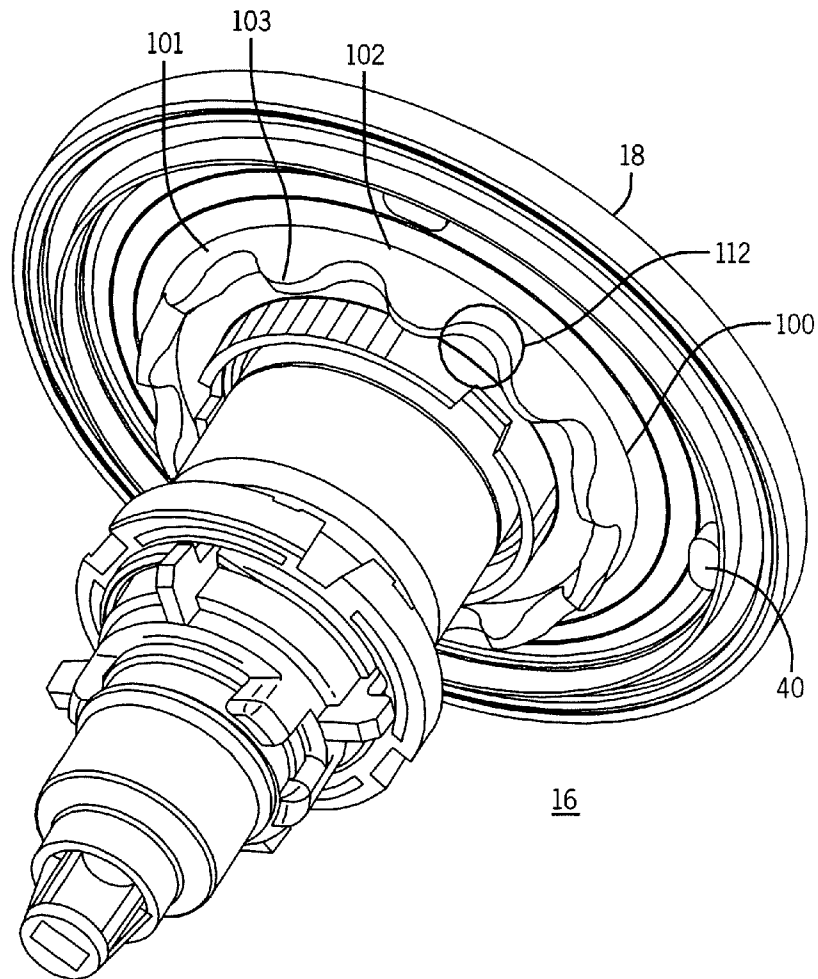


FIG. 4B

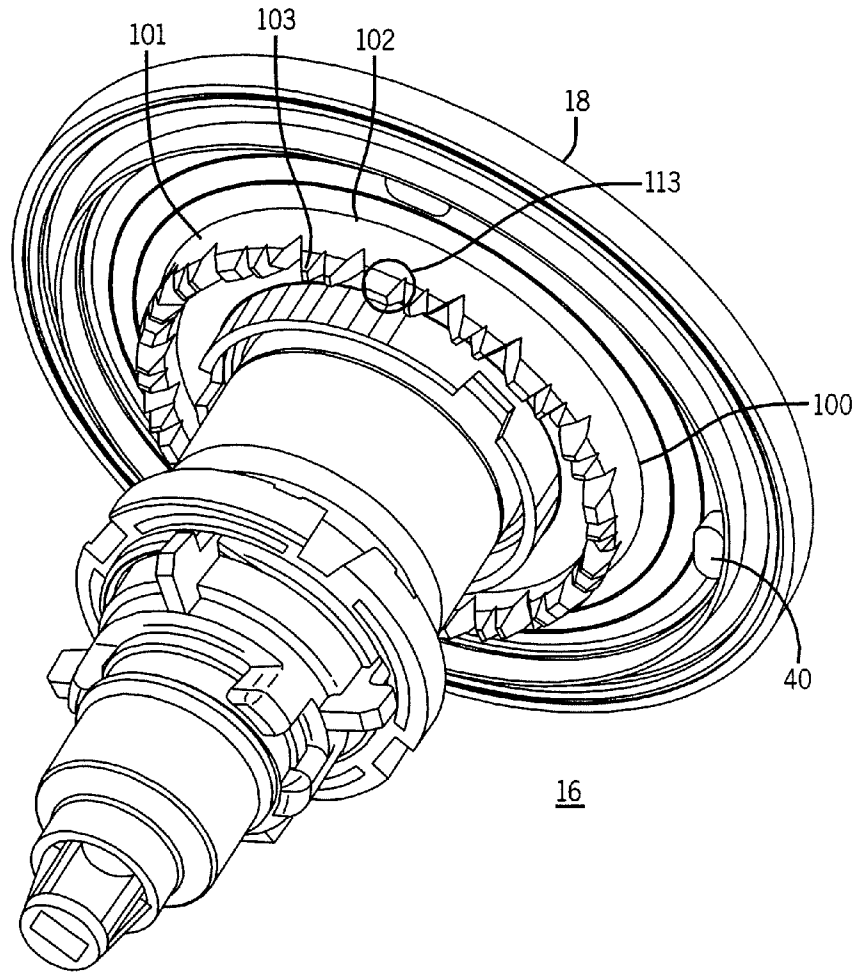


FIG. 4C

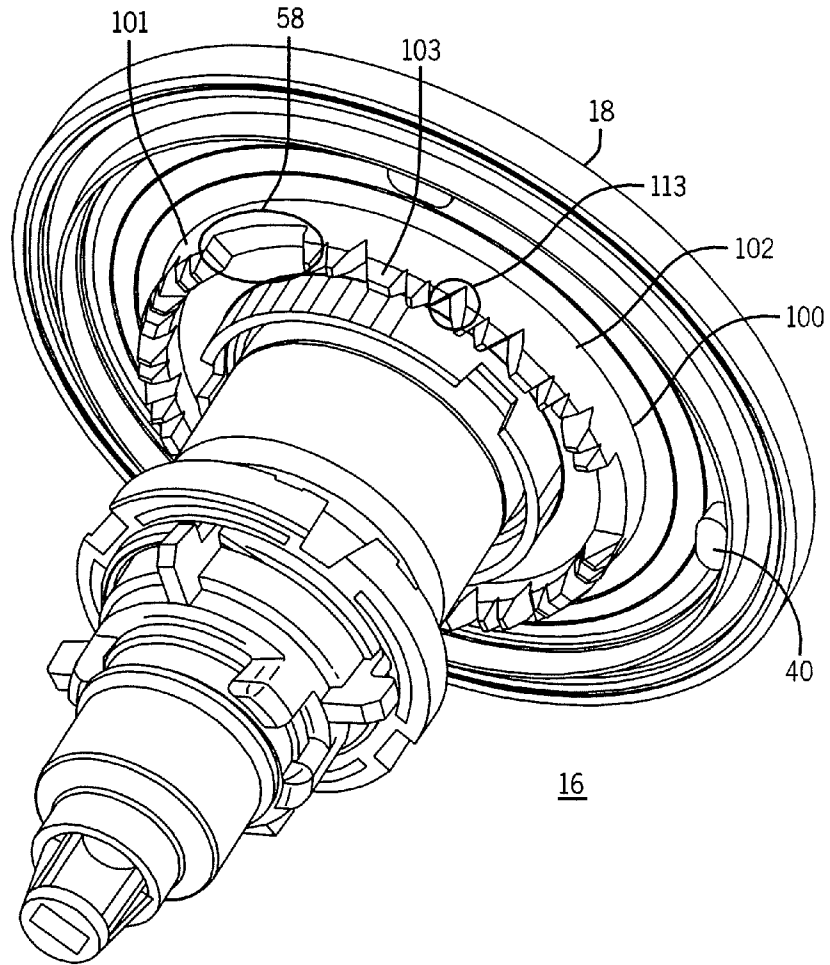


FIG. 4D

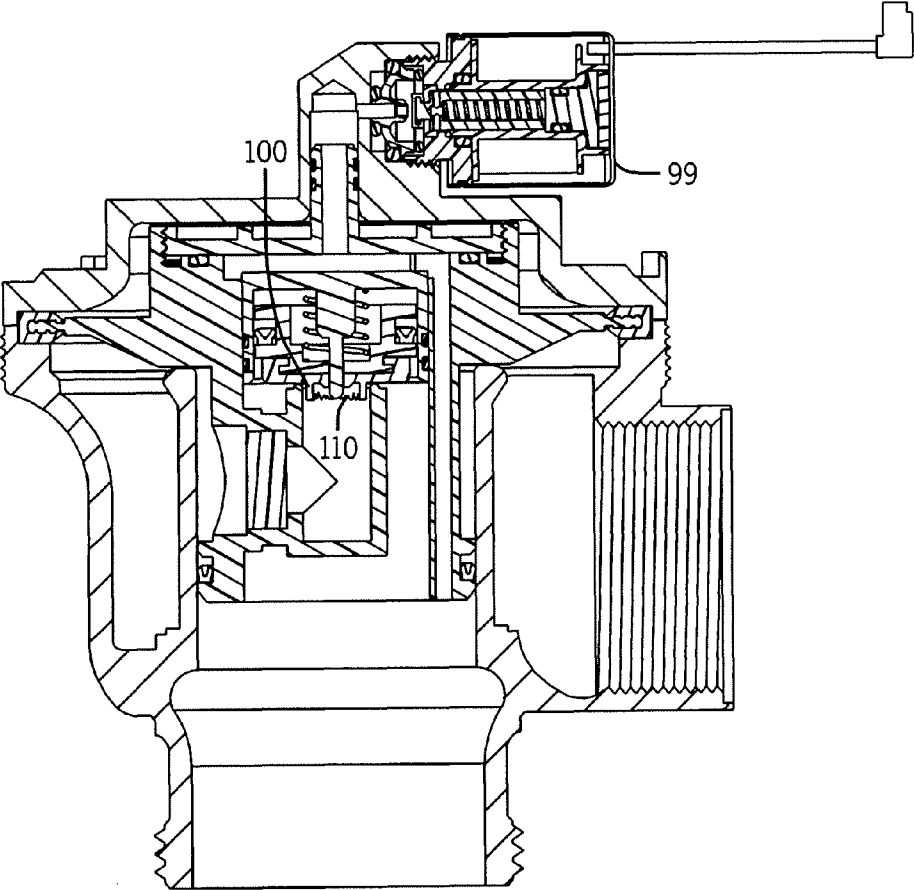


FIG. 5A

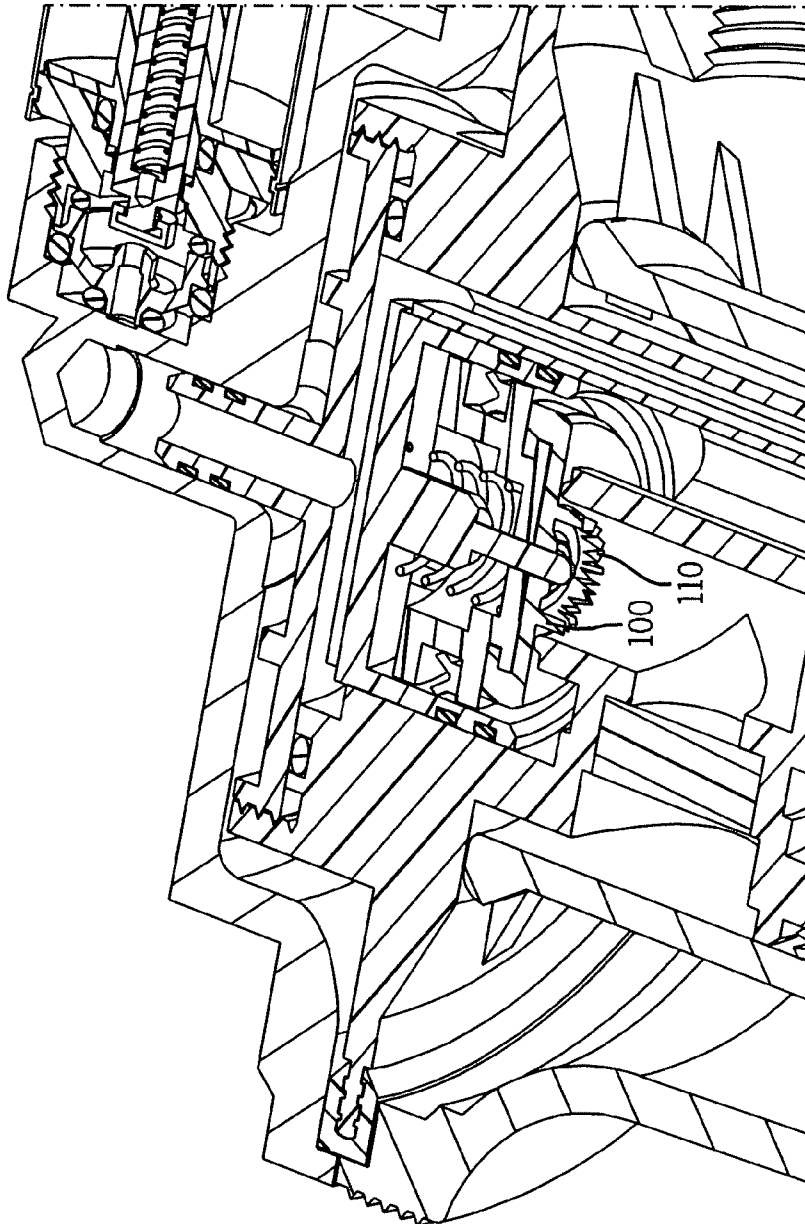


FIG. 5B

FIG. 6A

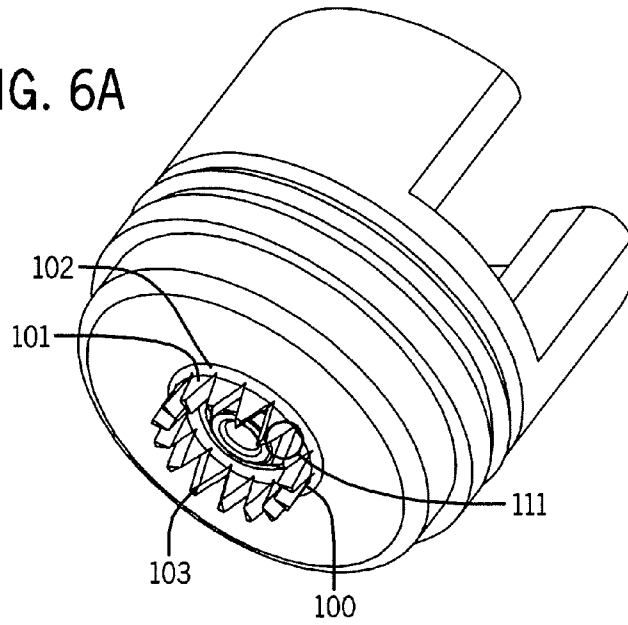


FIG. 6B

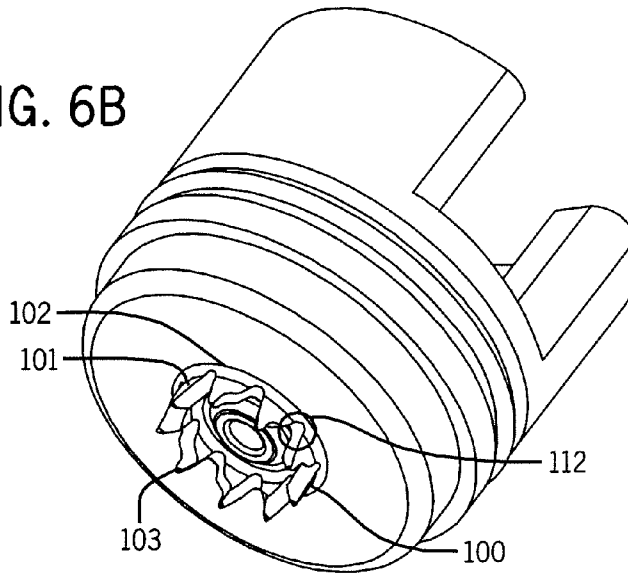


FIG. 6C

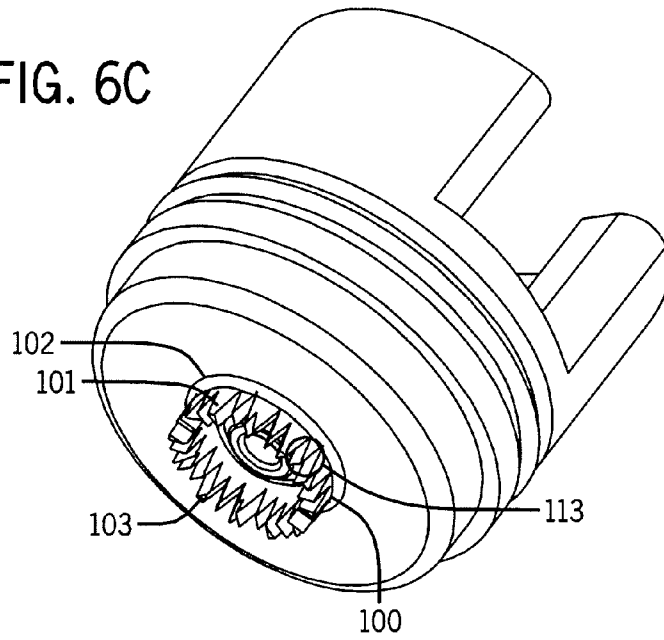
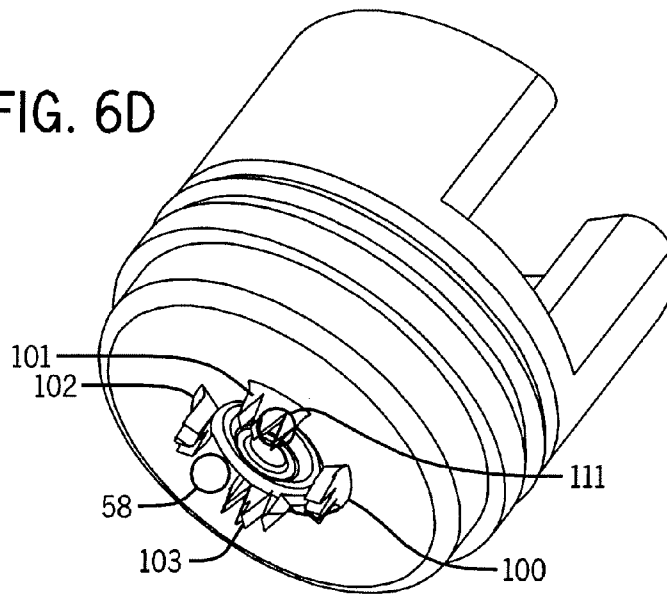


FIG. 6D



FLOW NOISE RESTRICTOR**CROSS-REFERENCE TO RELATED APPLICATIONS**

This application is related to and claims priority from U.S. Provisional Patent Application No. 61/542,666, filed Oct. 3, 2011 and is hereby incorporated by reference in its entirety.

FIELD OF THE INVENTION

The present invention generally relates to valves. More specifically, the present invention relates to flushometers or faucet valves having a flow noise restrictor.

BACKGROUND OF THE INVENTION

Numerous valves utilize a valve seat in their structure. Many of these structures have a valve structure that, typically, descends to seat upon the valve seat. Where there is a pressure differential between the area "upstream" of the valve seat and the area "downstream" of the valve seat, the energy of the system may be dissipated in undesirable ways. For example, cavitations and/or vibrations can occur, particularly at the moment the valve closes. These occurrences are often reflected in noise at the valve or its associated fixture or upstream/downstream due to vibrations traveling throughout the system. In liquid systems, the vibrations are sometimes caused by pressure waves traveling in the piping system that supplies the valve including what is commonly called water hammer. At other times the cause of the vibrations is more local due to gas trapped in the liquid providing an unstable dynamic system that tends to vibrate at certain flow conditions.

One particular type of valve that can exhibit "noise" problems is a flushometer, commonly used with water closets and urinals. Two particular types of flushometers are well known: diaphragm flushometers and piston flushometers. Diaphragm-type flushometers are exemplified by the flush valve shown in U.S. Pat. No. 6,616,119, which is hereby incorporated herein by reference. Piston-type flushometers are also known, as exemplified by the flush valve shown in U.S. Pat. No. 4,261,545, which is hereby incorporated herein by reference.

A flushometer or faucet valve includes a body **10** with an inlet **12** and outlet **14**, a valve assembly **15** with a valve seat **26**, a valve member **17** movable in the body **10** toward or away from the valve seat **26** to control flow from the inlet **12** to the outlet **14**. The valve assembly **15** has a pressure chamber **50** acting on one side of the valve member **17** opposing the inlet pressure on the other side of the valve member **17**. A bypass **40** connects the chamber **50** with the water inlet side. Pressure in the chamber **50** maintains the piston **80** or diaphragm **18** seated to the valve seat **26** and the valve assembly **15** in the closed position. There is a relief valve **30**, which may be a mechanical relief valve stem **32** or a solenoid **99** (FIG. 3A) driven, that vents the chamber **50** to the outlet **14** side of the valve to permit the piston **80** or diaphragm **18** to move away from the valve seat **26** and open and control the water flow thru the valve. The piston **80** or diaphragm **18** may have a portion **89/48** to keep it concentric to the valve seat **26** and in axial alignment with the valve seat **26**. The valve typically has a refill head **47** or similar flow control device on the outlet side of the diaphragm **18** or piston **80** to confine the path of flow. Valves of this kind are taught in prior art for example in U.S. Pat. Nos. 5,881,993; 5,887,848; 5,213,305; 5,244,179; 6,182,689; 6,260,576; 5,332,192 5,967,182.

It is well known, that in certain environmental and flow conditions, flushometers, such as those discussed above, can start to vibrate and cause noticeable and sometimes undesirable noise. Valve noise in the above described type of valves can be generated thru various mechanisms. If the pressure in some areas falls below vapor pressure due to the Bernoulli Effect, cavitation can occur, which can cause violent oscillations and forces on the valve. Air may become trapped or present in the air chamber, such as due to a high level of gas dissolved in the water from the inlet. Air entrapped in the pressure chamber **50** can introduce a different impedance, due to the variance in compression of the mixed air/water fluid compared to only water, of the piston/diaphragm and pressure chamber **50** and therefore make the flow unsteady. In addition, the piping upstream or downstream of the valve can cause undesirable oscillations in the valve.

This noise can also be described as flutter or water hammer. Numerous attempts have been made to address such noise. Some valves as described in U.S. Pat. No. 4,248,270 employ a resilient flow control device that deflects or deforms under the inlet pressure, and therefore dynamically controls the flow rate. U.S. Pat. No. 6,616,119 employs a diaphragm that has a molded rubber skirt on the inlet side of the flush valve which deforms with pressure and controls the flow. The skirt attempts to dampen vibration with "friction" tabs. The disadvantage of the resilient member often is that the modulus of elasticity of such members rapidly changes with temperature. It therefore makes it difficult to control the flow rates consistently over different operating temperatures due to the tabs' (of the '119 patent) friction against the outer diameter of the barrel.

Another means to control noise is to introduce friction between the moving diaphragm or piston and the valve housing. For example, U.S. Pat. No. 5,865,420 diaphragm teaches a refill head **47** on the outlet side of the valve which introduces friction between the housing and the moving refill head, therefore damping vibrations. The aforementioned refill head **47** on the inlet side also touches the housing barrel to introduce friction.

Some valves, e.g. U.S. Pat. No. 4,040,440 employ sound absorbing treatment on the outlet side, or generate turbulence as taught in U.S. Pat. No. 4,967,998. Some flushometer designs have grooves in the outlet skirt as well (made of plastic or metal) to control the flow as well. Other cage type valves employ perforated and grooved members, plugs and skirts as a means to make the flow turbulent to reduce noise throughout the flush cycle as shown in U.S. Pat. No. 4,024,891 or 3,990,475. However, the limited stroke of the chamber controlled valves does not allow for elaborate absorption treatment or perforation of members. In addition, the difficulty of those perforated and grooved members shown in prior art, is that even though they suppress noise thru the introduction of turbulence, they severely restrict flow thru the valve when the valve is in an open position or opening/closing stroke. In other configurations, the geometry adds friction or flow resistance to the opening or closing stroke. This cannot be adopted in valves that have a smaller stroke and larger flow rate requirements.

Further complicating matters, some of the portion of the noise/hammer occurs at the moment before the closing of the valve is completed. The Bernoulli effect is especially strong at that moment, as the inlet pressure builds up to static pressure of a typical residential or commercial water supply line, while at the same time the pressure on the outlet dramatically reduces (typical to atmospheric pressure). Present mechanisms at the outlet side of the valve seat have only little effect at that moment.

SUMMARY OF THE INVENTION

One embodiment of the invention relates to a flow noise restrictor having features for generating vortices.

One embodiment of the invention relates to a flush valve having a valve body having an inlet and an outlet. A valve assembly is included comprising a valve member and a valve seat. The valve member is seatable upon the valve seat to seal the inlet from the outlet. The valve assembly has a flow noise restrictor adjacent to the valve seat and partially defining a fluid flow path. The flow noise restrictor has a sidewall and a fluid flow edge defining a plurality of features.

One embodiment of the invention relates to a valve assembly comprising a valve member and a valve seat. The valve member is seatable upon the valve seat to seal the inlet from the outlet. The valve assembly has a flow noise restrictor adjacent to the valve seat and partially defining a fluid flow path. The flow noise restrictor has a sidewall and a fluid flow surface defining a plurality of features.

One embodiment of the invention relates to a flow noise restrictor for use with a valve assembly. The flow noise restrictor comprises a circular sidewall. The circular sidewall has an upper edge and a lower edge nonparallel with each other. One of the upper edge and lower edge configured to engage a portion of a flush valve. The other of the upper edge and lower edge define a plurality of features.

Additional features, advantages, and embodiments of the present disclosure may be set forth from consideration of the following detailed description, drawings, and claims. Moreover, it is to be understood that both the foregoing summary of the present disclosure and the following detailed description are exemplary and intended to provide further explanation without further limiting the scope of the present disclosure claimed.

BRIEF DESCRIPTION OF THE DRAWINGS

The foregoing and other objects, aspects, features, and advantages of the disclosure will become more apparent and better understood by referring to the following description taken in conjunction with the accompanying drawings, in which:

FIG. 1 is a side, partial sectional, view of a diaphragm flushometer.

FIG. 2 is a side, partial sectional, view of a piston flushometer.

FIG. 3A is a side, cross-sectional view of a diaphragm valve of one embodiment of the invention; FIG. 3B is a perspective, cross-sectional view of the diaphragm valve of FIG. 3A.

FIGS. 4A-4D illustrate diaphragm valve assemblies of various embodiments; FIG. 4A illustrates a flow noise restrictor having triangular features; FIG. 4B illustrates a flow noise restrictor having sinusoidal features; FIG. 4C illustrates a flow noise restrictor having irregular, sharp features; FIG. 4D illustrates a flow noise restrictor irregular, sharp features and large window openings.

FIG. 5A is a side, cross-sectional view of a piston valve of one embodiment of the invention; FIG. 5B is a perspective, cross-sectional view of the piston valve of FIG. 3A.

FIGS. 6A-6D illustrate piston valve assemblies of various embodiments; FIG. 6A illustrates a flow noise restrictor having piston features; FIG. 6B illustrates a flow noise restrictor having sinusoidal features; FIG. 6C illustrates a flow noise

restrictor having irregular, sharp features; FIG. 6D illustrates a flow noise restrictor irregular, sharp features and large window openings.

DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

In the following detailed description, reference is made to the accompanying drawings, which form a part hereof. In the drawings, similar symbols typically identify similar components, unless context dictates otherwise. The illustrative embodiments described in the detailed description, drawings, and claims are not meant to be limiting. Other embodiments may be utilized, and other changes may be made, without departing from the spirit or scope of the subject matter presented here. It will be readily understood that the aspects of the present disclosure, as generally described herein, and illustrated in the figures, can be arranged, substituted, combined, and designed in a wide variety of different configurations, all of which are explicitly contemplated and made part of this disclosure.

In some embodiments, the present invention relates to a flow noise restrictor **100** associated with a valve assembly **15** in a flushometer valve **1**. The flow noise restrictor **100** may have features **110** (such as regular triangular features **111**, sinusoidal features **113**, and irregular triangular features **113**), which create vortices between the valve member **17** and the valve seat **26** as the valve member **17** is being seated. The flow noise restrictor **100** narrows the inflow area as the valve assembly **15** closes. It should be appreciated that the water may flow through an area defined by the valve seat **26** and the flow noise restrictor **100**, with the features **110** of the flow noise restrictor **100** contributing to that area. As the distance between the flow noise restrictor **100** and valve seat **26** decreases during the valve assembly **15** closure, the percentage of the flow area contributed by the features **110** of the flow noise restrictor **100** increases. The features **110** of the flow noise restrictor **100** introduce larger scaled vortices that provide mixing of the fluid without significantly changing flow resistance in the open position or adding friction to the opening and closing stroke.

FIG. 1 illustrates a typical prior art diaphragm flushometer valve and FIG. 2 illustrates a typical prior art piston flushometer valve. The flush valve **1** includes a body **10** having an inlet **12** and an outlet **14** and a main valve seat **26** for sealing by a valve assembly **15**. A pressure chamber **50**, or electronic control mechanism (for example, as shown in FIG. 3A), is typically provided above the valve assembly **15**. This pressure chamber **50** may be pressurized by the line pressure of the inlet through bypasses **40** which place the pressure chamber **50** in fluid communication with the inlet **12**. The pressure chamber **50** is sealed from the outlet **14** by a relief valve **30** of the valve assembly **15**. The relief valve **30** includes a relief valve stem **32** extending downward through a relief valve seat **38** within the valve assembly **15** such that unseating of the relief valve **30**, such as by tilting the valve stem **32**, allows venting of the pressure chamber **50** to the outlet **14**. This reduces the pressure in the pressure chamber **50**, allowing the valve assembly **15** to be forced off of the main valve seat **26** by the pressure of the inlet **12**. The water from the inlet **12** may then pass through the main valve seat **26** to the outlet **14**. The valve assembly **15** reseats as the pressure chamber **50** reaches equilibrium pressure with the inlet forces acting on the valve assembly **15**.

With reference to FIG. 1 (diaphragm flushometer) and FIG. 2 (piston flushometer), the valve assembly **15** is actuated by an operating handle **22** which is fastened to the valve body **10**

5

by means of a coupling nut 23. The handle 22 is connected to a plunger 27 which extends to the interior portion of the valve body 10 below the main valve seat 26. As best shown in FIG. 2, the plunger 27 is guided and supported by a bushing 28 and is restored by a spring 25. A seal packing 33 may be snapped on the end of bushing 28 and prevents leakage outwardly from the handle opening. The valve 1 as shown in FIGS. 1 and 2 has a manual handle 22 for operation. The valve 1 is equally adaptable to automatic operation, for example by a solenoid 99 as set forth in U.S. Pat. No. 3,778,023, either by mechanized action on the handle 22 or an automatic actuation device directly interacting with the plunger 27 or relief valve stem 32.

With respect to FIG. 1, the valve assembly 15 of a diaphragm flushometer valve is a diaphragm assembly 16 that includes a diaphragm 18. In one embodiment, the diaphragm 18 is peripherally held to the body 10 by an inner cover 20. The diaphragm 18 is seated upon a shoulder 21 at the upper end of body 10 and is clamped in this position by the inner cover 20. An outer cover 24 is screw threaded onto the body 10 to hold the inner cover 20 in position.

The diaphragm assembly 16, as shown in the embodiment of FIG. 1, is closed upon a valve seat 26 formed at the upper end of a barrel 31. The barrel 31 forms the conduit connecting the valve seat with outlet 14. The diaphragm assembly 16 includes a relief valve 30 having a downwardly extending stem 32 carrying a movable sleeve 34. Sleeve 34 is positioned for contact by a plunger 27 when operated by a handle 22 as its is conventional in the operation of flush valves 1 of the type described.

In one embodiment, the diaphragm assembly 16, in addition to diaphragm 18 and the relief valve 30, includes a retaining disk 19, a refill ring 45 and a flow control ring 44. It should be appreciated that the diaphragm 18 may be a unitary component, such as described in U.S. Pat. No. 7,980,528, incorporated by reference herein. The underside of the retaining disk 19 is threadedly attached to a collar 46, which in turn is threadedly attached at its exterior to a sleeve 48 which carries the refill ring 45. The above described assembly of elements firmly holds the diaphragm 18 between the upper face of the refill ring 45 and a lower facing surface of the collar 46.

Above the diaphragm assembly 16 is the pressure chamber 50, which maintains the diaphragm assembly 16 in a closed position when the flush valve 1 is not in use. The pressure chamber 50 is fillable via the bypasses 40 and vents through the relief valve 30 into the barrel 31 and ultimately the outlet 14 of the flush valve 1.

As is known in the art, such as FIG. 1, when the handle 22 is operated, the plunger 27 will contact sleeve 34, lifting the relief valve 30 off its seat on the retaining disk 19. This will permit the 50 discharge of water within the pressure chamber 50 down through the sleeve 84. Inlet pressure will then cause the diaphragm 18 to move upwardly off its seat 26, permitting direct communication between the inlet 12 and the outlet 14 through the space between the bottom of the diaphragm assembly 16 and the seat 26. As soon as this operation has taken place, the pressure chamber 50 will begin to build through the bypass orifice 40 in the diaphragm assembly 16. As flow continues into the pressure chamber 50, the diaphragm assembly 16 will move toward its valve seat 26 and stop when it has reached that position, the flush valve 1 will be closed.

The diaphragm 18 of FIG. 1 has a peripheral edge 52 which will be held between the shoulder 21 of the body 10 and the inner cover 20. Spaced from the edge 52 is a downwardly extending rim 35, shown particularly in the section of FIG. 1.

6

When in the closed position, the rim 35 will extend about the upper end of the barrel 31. In one embodiment, the features 110 are sized on an order of magnitude relative to the distance of the stroke of the valve assembly 15.

Generally speaking, for a manual valve, the valve 1 is opened when an relief valve stem 32 is moved and opens a passage to the pressure chamber 50 above the diaphragm 18 or piston 80, and vents at least a portion the liquid to the outlet 14 side of the valve 2, therefore lowering the pressure above the diaphragm 18 or piston 80 and allowing the pressure below the diaphragm 18 or piston 80 to move the respective diaphragm 18 or piston 80, thus opening the valve. For embodiments using an automatic actuation mechanism that triggers the plunger 27, a similar process occurs. For embodiments utilizing a separate actuation mechanism from the traditional handle 22, such as utilizing a solenoid, the pressure chamber 50 above the diaphragm 18/piston 80 is opened by electronic means such as a latching solenoid valve 99, draining said cavity to the outlet side of the valve 1, allowing the piston 80/diaphragm 18 to move to the open position. Various automatic or manual actuation systems are known in the art and may be used without departing from embodiments of the present invention.

In one embodiment, the flow noise restrictor 100 may be used with a piston flushometer having piston assembly 79. A piston assembly 79 indicated generally at 34 is adapted to reciprocate within the body 10. Although one embodiment of a piston assembly 79 is described below, it should be appreciated that the various types of piston assemblies may be used without departing from the present invention. The piston assembly 79 includes a hollow, generally cylindrical piston 80. The piston 80 has a lower cylindrical portion 89 which is directly adjacent a piston seat area 73, with the seat area 73 being normally seated upon a seal 83 to close the main valve seat 26 and to thereby control the flow of water through the flushometer valve 1.

The piston 80 of FIG. 2 has a pair of bypass orifices 40, which are illustrated with an optional filter ring 43, which ring 43 functions according to known principles for providing additional anti-clogging properties. The interior chamber 42 of the piston 80 has a relief valve seat 38, which may include a seal 83. The seat 38 and seal 83 are at the top of a central passage which connects chamber 42 with the outlet 14 side of the flushometer valve 1.

The piston assembly 79 also includes a relief valve 30 which normally closes the piston 80. The relief valve 30 has a shoulder 49 which engages the seal 83. An operating stem 32 is slidable in the interior chamber 42 of the relief valve 30 and extends to a point adjacent plunger 27. A spring 85 assists in holding the relief valve 30 in its position to close and seal chamber 42.

The piston assembly 79 further includes a cap 86 threadedly engaging the upper wall of piston 80. The cap 86 has a central stop 87 against which the spring 85 abuts. The stop has holes 88 which provide fluid communication between the piston interior chamber 42 and an upper pressure chamber 50. A packing member or seal member 64 held between the cap 86 and piston 80 provides a slidable seal separating the pressure chamber 50 from the inlet's 12 water pressure except through the bypass 43.

The piston 80 has a cylindrical wall 70 which is preferably smooth and unobstructed. Directly adjacent the cylindrical wall 70 is a tapered piston area 72 which may have a taper of on the order of about ten degrees, which taper is effective to provide a clear flow path about the piston when it is in the raised position away from the valve seat 26. Directly adjacent the tapered area 72 is the piston seat area 73 which will close

upon the seat **26** when the valve is in the closed position. Directly downstream of the piston seat area **73** is a ring **74** which has an outer diameter slightly less than the diameter of the valve outlet adjacent the seat **26** so that ring area **74** will be inside of the valve seat **26** when the piston **80** is closed. The ring **74** functions as a throttling means in that it substantially reduces flow through the valve outlet just prior to complete valve closure.

Directly adjacent the throttling ring **74** is cylindrical portion **89** which has a plurality of radially and axially extending ribs **76**. The outer diameter of the ribs **76** is less than wall **70** and just slightly less than the passage through seat **26**. The ribs **76** are thus inside of the major portion of the piston **80** so as not to restrict flow. In a preferred embodiment five ribs **76** are provided for maximizing stability and guidance for the piston **80**, without detrimentally obstructing water flow past the piston **80** when the piston **80** is in the valve open position. At the lower end of each of the axially extending ribs there is a chamfered area **78** which assists in assembling the piston **80** within the flushometer valve **1**.

The area between each of the circumferentially, generally uniformly spaced ribs **76** is closed by a skirt **90**. As shown, the skirt **90** has a radius slightly less than the exterior surface of the ribs **76**. The function of the skirt **90** is to close the area between ribs to provide control of water flow past the piston **80**, which in turn will provide a more consistent operation of the flushometer. The skirt **90** improves the flow path by maintaining it in an axial direction generally circumferentially about the cylindrical piston portion **89**. By preventing water flow into the sleeve **48**, the skirt **90** also helps prevent any back pressure which might retard closure of the relief valve **30**.

Typically, during the flush cycle, the water below the valve assembly **15** and passing over the main valve seat **26** exhibits generally laminar flow. The present invention relates to the suppression of noise in valve assemblies. In one embodiment, the present invention suppresses noise at the closing of the valve by causing a maximized pressure drop right before valve closure and also introducing vortices (whirling flow+turbulent flow) into the flow to stabilize the flow across the valve seat **26**. Whirling flow suppresses noise while at the same time not restricting the flow.

In one embodiment, the invention may be in a form of a flow noise restrictor **100** at the inlet side of the valve. The flow noise restrictor **100** may be a portion of the rim **35**, with a series of regular or irregular features **100**. In some embodiments, the flow noise restrictor **100** is near to the valve seat **26**, such as adjacent the valve seat **26**. FIGS. 3A-4D illustrate a diaphragm assembly **16** flushometer valve **1** having a flow noise restrictor **100** of the present invention. FIGS. 5A to 6D illustrate a solenoid controlled piston assembly **78** flushometer valve **1** having a flow noise restrictor **100** of the present invention.

In some embodiments the flow noise restrictor **100** includes a sidewall **101**, a valve assembly surface **102** adjacent a portion of the valve assembly, for example abutting against the diaphragm **18**, and a flow surface **103**, which may be an edge or face of the sidewall **101**. In one embodiment, the flow noise restrictor **100** has a ring-like shape. The flow surface **103** is defined by a plurality of features in the sidewall **101**. In one embodiment, the flow surface **103** is non-parallel with the valve assembly surface **102** and/or the valve member **17**.

It is not necessary, in one embodiment, for the flow noise restrictor **100** to be in direct contact with the valve seat **26** or the valve body **10**. The flow noise restrictor **100** controls the flow through the valve **1** by restricting the area through which

the flow may pass. In one embodiment, more as the piston **80** or diaphragm **18** reaches the valve seat **26** and less as the valve fully opens. In certain embodiments, the flow noise restrictor **100** has proportions relative to the other components of the flushometer valve **1** and particularly relative to the area through which the water flows during a flush cycle that allow for the impact of the induced vortices to become meaningful just prior to the valve **1** closing. It should be appreciated that such allows for unhindered flow when the valve **1** fully opens, but creates an increasing pressure drop before flow reaches the valve seat **26** while closing. This is achieved by giving the flow noise restrictor **100** a variable circumferential cross section, where the cross sectional area (allowing for water flow, such as the area of windows **58**) increases in the direction of the opposing portion of the valve assembly **15**.

In one embodiment, the flow noise restrictor **100** comprises a series of features **110**, which may have one or more associated shape (such as triangular **111**, sinusoidal **112**, or irregular triangular **113**). For embodiments as discussed where the cross sectional area increases, these features **110** have, on average, a decreasing width from a portion of the feature closest the valve assembly surface **102** of the flow noise restrictor **100**. It should be appreciated that individual features **110** may have an inverted shape where the width decreases but the overall total width of all features **110** increases. The features **110** may be such that the flow surface **103** defines a side of the sidewall **101**.

In one embodiment, at least a portion of the features **110** are holes in the flow noise restrictor **100**. For embodiments where the cross sectional area increases, the area of the holes and/or the number of holes may increase towards the flow surface **103** of the flow noise restrictor **100**. For each particular annular cross-sectional slice or plane, the features define a width. The width of the features may vary in each cross-sectional slice. For example, the width may increase as one proceeds from the valve assembly surface **102** to the flow surface **103**.

In one embodiment, the flow noise restrictor **100** includes a sidewall **101** that is either curved, curvilinear, or a series of linear edges. The height of the sidewall **101**, i.e. the distance between the flow surface **103** and the valve assembly **15** may vary. In one embodiment, the features **110** are symmetrical. In another embodiment, the features **110** are nonsymmetrical, i.e. irregular.

In one embodiment illustrated in FIG. 4A (diaphragm) and FIG. 6A (piston flushometer), this is achieved with triangular shaped features **111** in the sidewall **101** of the flow noise restrictor **100**, which is spaced about the circumference of the valve seat **26**. The triangular pattern around the circumference could be replaced with other geometries such as semi circular or sinusoidal **112** (FIG. 4B (diaphragm) and FIG. 6B (piston flushometer)), as long as it has a large base area that gets gradually smaller away from flow surface **103**. In addition, the features **110** may be irregular such as FIG. 4C (diaphragm) and FIG. 6C (piston flushometer).

In one embodiment, there are at least three features **110** along the circumference of the valve seat. In one embodiment, as many features **110** as feasible are provided to introduce multiple three-dimensional vortical flow structures into the inlet flow to the valve seat **26**, with the hi and low differential in the geometry of the flow noise restrictor **100**. However, the circumferential structures/features need to stay large enough to influence the flow gradually over a larger part of the closing stroke of the diaphragm **18**/piston **80**. In addition, in one embodiment, the features **110** of the flow noise restrictor **100** may be sharp features to add small scale turbulent structures to the inflow of the valve seat **26** geometry. In one

embodiment, the features **110** are substantially evenly spaced annularly about the flow noise restrictor **100**.

In the preferred embodiment the flow noise restrictor **100** on the inlet side of the valve seat **26** is combined with an existing noise reduction design such as methods of friction and flow restriction. This allows for two steps of pressure reduction between the inlet **12** and outlet **14** of the closing valve **1** and therefore minimizes the possibility of cavitation noise. First, the flow noise restrictor **100** provides for a pressure reduction. Second, the refill ring **45** (e.g. for a diaphragm flushometer) provides a reduction in pressure on the outlet **14** side. The use of the flow noise restrictor **100** allows for the suppression of noise (vibration) purely by manipulating the flow.

In one embodiment, the flow noise restrictor **100** is positioned adjacent either the valve seat **26** or the piston **80**/diaphragm **18**. In an embodiment, the flow noise restrictor **100** is mounted on a moving member of the valve **1**, such as the diaphragm **18** or piston **78**, but an alternate method can be envisioned where the flow noise restrictor **100** can be part of the valve housing or made of two parts, one part being attached to the housing or valve seat and one attached to the diaphragm **18**/piston **80**. In one embodiment, a piston flushometer includes the flow noise restrictor positioned about the periphery of the piston **80**. In one embodiment, at the tapered area **72**. In a further embodiment, the flow noise restrictor **100** includes a portion about the periphery of the piston **80** and a corresponding portion engagable therewith on the valve seat **26**.

In one embodiment, the downwardly extending rim **35** comprises the flow noise restrictor **100**, with the features **110** being defined by portions of the rim **35** that are removed.

In one embodiment, the features may include one or more windows **58**. As shown in FIGS. 4D and 6D, wherein the flow noise restrictor **100** includes a plurality of windows **58** which will modulate the flow of water as the diaphragm **18** (or piston **80**) closes upon the valve seat **26** at the upper end of the barrel **31**. The windows **58** provide a uniform shape that does not alter the flow area as the valve closes, i.e. the cross-sectional area remains the same during closing. The windows **58** are significant openings in the flow noise restrictor **100** sidewall **101** when compared to the size of the individual features **110**. The window's **58** geometry size and shape is not small enough to add sufficient vorticity to suppress noise.

In one embodiment, the features **110** are molded into the flow noise restrictor **100**. In another embodiment, the flow noise restrictor **100** is cut/drilled to form the features **110**.

In one embodiment, the flow noise restrictor **100** does not contact the barrel **31** or valve seat **26**.

The foregoing description of illustrative embodiments has been presented for purposes of illustration and of description. It is not intended to be exhaustive or limiting with respect to the precise form disclosed, and modifications and variations are possible in light of the above teachings or may be acquired from practice of the disclosed embodiments. It is intended that the scope of the invention be defined by the claims appended hereto and their equivalents.

What is claimed is:

1. A flush valve comprising:
 - valve body having an inlet and an outlet
 - a valve assembly comprising a valve member and barrel in communication with the inlet and the outlet and having an exterior surface and an interior surface with a valve

seat defined at the upper portion of the barrel, the valve member seatable upon the valve seat to seal the inlet from the outlet;

the valve assembly having a flow noise restrictor adjacent to the valve seat; and

the flow noise restrictor having a sidewall with a plurality of features formed in the sidewall and partially defining a fluid flow surface non-parallel with the valve member, the flow noise restrictor disposed about the exterior portion of the barrel and the fluid flow path extending from exterior of the flow noise restrictor, through the plurality of features to interior of the flow noise restrictor and into the barrel;

wherein the flow noise restrictor has a continuously variable circumferential cross-section.

2. The flush valve of claim 1, wherein the flow noise restrictor is positioned such that vortices in fluid flowing through the flush valve are positioned upstream of the valve seat.

3. The flush valve of claim 1, wherein the sidewall is curved.

4. The flush valve of claim 1, wherein the flush valve assembly comprises a diaphragm assembly and the valve member comprising a diaphragm wherein the flow noise restrictor extends from the diaphragm.

5. The flush valve of claim 1, wherein the plurality of features are selected from the group consisting of sinusoidal, triangular, and semi-circular.

6. The flush valve of claim 1, wherein the flow noise restrictor comprises angled edges.

7. A valve assembly comprising:

a valve member and a valve seat, the valve member seatable upon the valve seat to seal the inlet from the outlet; and

the valve assembly having a flow noise restrictor adjacent to the valve seat and partially defining a fluid flow path; and

the flow noise restrictor having a sidewall and a fluid flow surface defining a plurality of features;

wherein the flow noise restrictor has a continuously variable circumferential cross-section; and

further wherein the flow noise restrictor features extend from the valve member towards the valve seat and the fluid flow path passes between each of the features perpendicular to the features, the flow noise restrictor at least partially annularly disposed about an exterior circumference of the valve seat when the valve assembly is closed.

8. The valve assembly of claim 7, wherein the flow noise restrictor is positioned such that vortices in fluid flowing through the valve assembly are positioned upstream of the valve seat.

9. The valve assembly of claim 7, wherein the sidewall is curved.

10. The valve assembly of claim 7, wherein the plurality of features are selected from the group consisting of sinusoidal, triangular, or semi-circular.

11. The valve assembly of claim 7, wherein the plurality of features each have a width increasing towards the fluid flow surface.

12. The valve assembly of claim 7, wherein the flow noise restrictor comprises angled edges.

13. The valve assembly of claim 7, wherein the flow noise restrictor features are non-symmetrically disposed about the flow noise restriction with respect to each other.