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SUSPENSION STRUCTURE AND DRIVING ASSEMBLY COMPRISING THE SAME**BACKGROUND OF THE INVENTION**Field of the Invention

5 The present invention relates to a suspension structure, and more particularly, to a suspension structure which is disposed between a driving wheel and a power generating device of a vehicle. Therefore, the suspension structure is adapted to absorb an impact force generated by the environment while the vehicle driving.

10 Descriptions of the Related Art

 In the time that mechanical technology has not been universal, the vehicle used to transport the goods was moved by human or animal drag. However, along with the development of technology, the movement for the vehicle is gradually changed into mechanical power.

15 Therefore, along with the advent of the mechanical technology age, the requirement for the vehicle is distinct from the previously age. For example, in the past, the vehicle is merely required if it can finish the transport operation during the working time. However, the vehicle is not only required to finish the transport operation during the working time, but also required to finish the transport operation with lowest power wastage or the most effective
20 way nowadays.

 What is more, in addition to the above requirements, it is also a big challenge about how

to maintain the stability while driving, so that even if the vehicle drives on the bumpy roads, the goods and the passengers placed on the vehicle will have no affected. Along with the aforementioned demand, the suspension structure used for absorbing the impact force to maintain the stability of the vehicle while driving is started with vigorous development.

5 The suspension structure used in the vehicle such as car or scooter nowadays, are disposed between a power generating device of the vehicle and a driving wheel. Thus, while the vehicle driving and the driving wheel is driven with a transmission element (e.g.: a belt) by the power generating device (e.g.: an engine) disposed on the vehicle, if the driving wheel is received an impact force by the bumpy road, then the impact force will be absorbed by the
10 suspension structure. That is, the vehicle will be able to provide a comfortable traveling quality for the goods or the passengers by the arrangement of the suspension structure.

However, since the suspension structure is able to absorb the kinetic energy of the impact force by transforming the kinetic energy to the potential energy, the potential energy will change the relative positions of the elements of the suspension structure. Therefore, the
15 distance between the power generating device of the vehicle and the driving wheel will be changed during the process of absorbing the impact force, and the transmission elements (i.e.: the belt) will become tight or loose, thereby seriously affecting the life of the transmission elements, and causes the low power transmission efficiency between the power generating device and the driving wheel.

20 Accordingly, an urgent need exists in the art to provide a suspension structure and a driving assembly comprising the same which are able to absorb the impact force generated by

the environment, and are able to minimize the changes in the distance between the power generating device and the driving wheel.

SUMMARY OF THE INVENTION

5 An objective of the present invention is to provide a suspension structure disposed between a power generating device of a vehicle and a driving wheel, wherein the changes in the distance between the power generating device and the driving wheel are able to be minimized by the arrangement of the suspension structure. Therefore, the tension of a transmission element disposed between the power generating device and the driving wheel
10 will keep a constant value, or change within a range of values.

To achieve the aforesaid objective, a suspension structure with a first suspension arm, a second suspension arm, a third suspension arm and a shock absorber component is provided. The first suspension arm has a first fixed end and a first swing end opposite to the first fixed end. The second suspension arm has a second fixed end and a second swing end opposite to
15 the second fixed end. The third suspension arm has a front swing end and a rear swing end opposite to the front swing end. The shock absorber component is disposed between the first suspension arm and the second suspension arm, and has two ends pivotally connected to the first suspension arm and the second suspension arm respectively. The shock absorber component is adapted to absorb an impact force by deforming while the impact force
20 generated.

The detailed technology and preferred embodiments implemented for the subject

invention are described in the following paragraphs accompanying the appended drawings for people skilled in this field to well appreciate the features of the claimed invention.

BRIEF DESCRIPTION OF THE DRAWINGS

5 **FIG. 1** is a schematic view illustrating a driving assembly with a suspension structure of the present invention;

FIG. 2 is a schematic view illustrating a driving assembly of the present invention driving on the road;

FIG. 3 is a schematic view illustrating the driving wheel of the driving assembly of the
10 present invention;

FIG. 4 is a schematic view illustrating the other embodiment of the suspension structure of the present invention; and

FIG. 5 is a schematic view illustrating a first axial direction, a second axial direction and a third axial direction of the suspension structure of the present invention.

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DESCRIPTION OF THE PREFERRED EMBODIMENT

With reference to both **FIG. 1** and **FIG. 2**, a driving assembly **100** of the present invention utilized in a vehicle **500** is provided, and the driving assembly **100** is adapted to provide assistance to the vehicle **500** driving on the road.

20 As shown in **FIG. 1**, the driving assembly **100** comprises a driving wheel **200**, a power generating device **300** and a suspension structure **400**. The power generating device **300** is

disposed on a driving position **510** of the vehicle **500** and connected to the driving wheel **200** by a transmission element **310** to provide power to the driving wheel **200**.

It is noted that the driving position **510** where the power generating device **300** disposed, is fixed on the vehicle **500**. In other words, no matter what kind of road the vehicle **500** drives on, the power generating device **300** is always disposed on the same position.

The suspension structure **400** comprises a first suspension arm **410**, a second suspension arm **420**, a third suspension arm **430** and a shock absorber component **440**, thereby the suspension structure **400** could better stable suspension for the vehicle **500** than prior art. In addition, with reference to **FIG. 3**, the driving wheel **200** comprises a rotating axis **210** on which the third suspension arm **430** is pivoted.

Hereinbelow, the arrangement between the first suspension arm **410**, the second suspension arm **420**, the third suspension arm **430** and the shock absorber component **440** of the suspension structure **400** of this embodiment will be further introduced.

As shown in **FIG. 1**, the first suspension arm **410** has a first fixed end **412** and a first swing end **414** opposite to the first fixed end **412**, wherein the first fixed end **412** is pivoted to the vehicle **500** at a first pivot point **416**. The second suspension arm **420** has a second fixed end **422** and a second swing end **424** opposite to the second fixed end **422**, wherein the second fixed end **422** is pivoted to the vehicle **500** at a second pivot point **426**. The third suspension arm **430** has a front swing end **432** and a rear swing end **434** opposite to the front swing end **432**, wherein the front swing end **432** is pivotally connected to the first swing end **414** of the first suspension arm **410** at a first swing point **436**, and the rear swing end **434** is

pivotally connected to the second swing end **424** of the second suspension arm **420** at a second swing point **438**. The shock absorber component **440** is disposed between the first suspension arm **410** and the second suspension arm **420**, and has two ends **442** pivotally connected to the first suspension arm **410** and the second suspension arm **420** respectively.

5 Through the above arrangement, the first suspension arm **410**, the second suspension arm **420**, the third suspension arm **430** and the vehicle **500** are generally presented a four-bar linkage structure. Besides, as the shock absorber component **440** is disposed between the first suspension arm **410** and the second suspension arm **420**, when the vehicle **500** is driving, the suspension structure **400** is adapted to absorb an impact force generated by the environment by transforming the kinetic energy of the impact force to the potential energy, wherein the potential energy will cause changes in the relative positions of the first suspension arm **410**, the second suspension arm **420**, the third suspension arm **430** and the shock absorber component **440**. On the other hand, as the suspension structure **400** is generally presented a four-bar linkage structure, this also benefits to limit the relative position between the first suspension arm **410**, the second suspension arm **420**, the third suspension arm **430** and the shock absorber component **440**. The suspension structure **400** is therefore to absorb the impact force generated by the environment with the minimum displacement, and maintain the stability of the vehicles **500**.

As shown in **FIG. 1**, the third suspension arm **430** further comprises a wheel axle point **435** disposed between the front swing end **432** and the rear swing end **434**. Therefore, when the rotating axis **210** of the driving wheel **200** is disposed on the wheel axle point **435**, the

driving wheel **200** is disposed on a position between the front swing end **432** and the rear swing end **434** of the third suspension arm **430**. As such, the distance between the driving position **510** and the rotating axis **210** is equal to the distance **d** between the driving position **510** and the wheel axle point **435** of the third suspension arm **430**.

5 Therefore, when the vehicle **500** with goods or passengers is driving on a flat road, the interaction between the first suspension arm **410**, the second suspension arm **420**, the third suspension arm **430** and the shock absorber component **440** will make the suspension structure **400** in a stable force mode. As a result, the weights of the vehicle **500** and the goods are uniform downward. The flat road will cause the driving wheel **200** with a force
10 uniform upward. That is, when the vehicle **500** drives on the flat road, the suspension structure **400** will have the illustration as shown in **FIG. 2**. In this moment, as the driving wheel **200** is with a force uniform upward, the shock absorber component **440** is in a compressed mode. The position where the shock absorber component **440** is pivoted to the
15 second swing end **424** of the second suspension arm **420** will slightly move upward as shown by arrow **A**. In such a case, the suspension structure **400** is not only used for absorb the impact force generated by the environment, but the wheel axle point **435** of the third suspension arm **430** also keeps a constant distance **d** to the driving position **510**. As the wheel axle point **435** of the third suspension arm **430** keeps a constant distance **d** to the driving position **510**, the tension of the transmission element **310** also keeps a constant value.

20 Therefore, when the vehicle **500** drives on a flat road, the suspension structure **400** will prolong the life of the transmission element **310**, and ensure the stability and comfort of the

vehicle **500** while driving.

On the other hand, when the vehicle **500** with goods or passengers drives on a bumpy road, since the bumpy road will make the driving wheel **200** in an impact state or a non-impact state continuously and irregularly, the suspension structure **400** is accordingly
5 compressed or relaxed. The position where the shock absorber component **440** is pivoted to the second swing end **424** of the second suspension arm **420** will not be in an equilibrium position, but be moved upward or downward. In detail, as shown in **FIG. 2**, the position where the shock absorber component **440** is pivoted the second swing end **424** of the second suspension arm **420** will be moved between the arrow **A** and the arrow **B**. The suspension
10 structure **400** is therefore swinging upward or downward, and changing the position corresponding to the bumpy road.

However, even though the vehicle **500** with goods or passengers drives on the bumpy road, and therefore cause the changes in the relative positions between the first suspension arm **410**, the second suspension arm **420**, the third suspension arm **430** and the shock absorber
15 component **440** of the suspension structure **400**, the distance **d** between the wheel axle point **435** of the third suspension arm **430** and the driving position **510** still changes within a range of values. Since the distance **d** between the wheel axle point **435** of the third suspension arm **430** and the driving position **510** changes within a range of values, the tension of the transmission element **310** changes within a range of values. Meanwhile, the change in the
20 tension of the transmission element **310** is minimized.

As a result, although the tension of the transmission element **310** will changes irregularly

when the vehicles **500** drives on the bumpy road with the suspension structure **400**, the change in the tension of the transmission element **310** is minimized effectively by the distinguished arrangement of the first suspension arm **410**, the second suspension arm **420**, the third suspension arm **430** and the shock absorber component **440**. The above
5 distinguished arrangement also prolongs life of the transmission element **310**, and ensures the stability and comfort of the vehicles **500** while driving.

After indicating the distinguished arrangement of the first suspension arm **410**, the second suspension arm **420**, the third suspension arm **430** and the shock absorber component **440**, the effect of the tension of the transmission element **310** and the stability and comfort of
10 the vehicles **500**, the followings are going to discuss the situation about the changes of the distance **d** between the wheel axle point **435** of the third suspension arm **430** and the driving position **510**.

First, if we define the position where the wheel axle point **435** disposed is moved within an allowable moving area while the driving wheel **200** is impacted but is not impacted by the
15 impact force generated by the environment, and the highest point of the allowable moving area is defined by the arrow **A**, and the lowest point of the allowable moving area is defined by the arrow **B**, then we can therefore define that the distance **d** between the wheel axle point **435** and the driving position **510** is changed within the range of values **D-D'**, wherein the value of **D'** is larger than the value of **D**.

20 As a result, with properly adjusting the relative arm lengths of the first suspension arm **410**, the second suspension arm **420** and the third suspension arm **430**, the change in the value

of the distance **d** between the wheel axle point **435** and the driving position **510** is varied correspondingly.

For example, when the relative arm lengths of the first suspension arm **410**, the second suspension arm **420** and the third suspension arm **430** are adjusted to cause the driving wheel **200** to be suspended (i.e. the driving wheel **200** is not impacted by the impact force generated by the environment, and the shock absorber component **440** is disposed on the arrow **B**), the distance **d** between the wheel axle point **435** and the driving position **510** is in the value of **D** (i.e. the distance **d** is minimized). Also, when the driving wheel **200** is impacted by the impact force generated by the environment, and the shock absorber component **440** is disposed on the highest point of the allowable moving area (i.e. the arrow **A**), the distance **d** between the wheel axle point **435** and the driving position **510** is in the value of **D'** (i.e. the distance **d** is maximized). As a result, in this first embodiment, the tension of the transmission element **310** is changed from loose to tight smoothly.

Further, if the relative arm lengths of the first suspension arm **410**, the second suspension arm **420** and the third suspension arm **430** are similarly adjusted to cause the driving wheel **200** to be suspended (i.e. the driving wheel **200** is not impacted by the impact force generated by the environment, and the shock absorber component **440** is disposed on the arrow **B**), the distance **d** between the wheel axle point **435** and the driving position **510** is in the value of **D'** (i.e. the distance **d** is maximized). Also, when the driving wheel **200** is impacted by the impact force generated by the environment, and the shock absorber component **440** is disposed on the highest point of the allowable moving area (i.e. the arrow **A**), the distance **d**

between the wheel axle point **435** and the driving position **510** is in the value of **D** (i.e. the distance **d** is minimized). As a result, in this second embodiment, the tension of the transmission element **310** is changed from tight to loose smoothly, contrary to the first embodiment.

5 In another possible embodiment, the relative arm lengths of the first suspension arm **410**, the second suspension arm **420** and the third suspension arm **430**, are properly adjusted to cause the distance **d** between the wheel axle point **435** and the driving position **510** is in the value of **D'** (i.e. the distance **d** is maximized) while the shock absorber component **440** is disposed between the highest point (i.e. the arrow **A**) and the lowest point (i.e. the arrow **B**) of
10 the allowable moving area. In addition, cause the distance **d** between the wheel axle point **435** and the driving position **510** to be in the value of **D** (i.e. the distance **d** is minimized) while the shock absorber component **440** is disposed on the highest point (i.e. the arrow **A**) or the lowest point (i.e. the arrow **B**) of the allowable moving area. As a result, in this third
15 loose.

The above statements for the change in the value of the distance **d** between the wheel axle point **435** and the driving position **510** describe the tiny changes in the tension of the transmission element **310** in different situations. However, it is noted that even though the tension of the transmission element **310** is changed in the above situations, the change in the
20 value is still far less than that in the prior art. Thus, the arrangement of the suspension structure **400** will prolong life of the transmission element **310**.

In order to make the tension of the transmission element **310** in a constant value, or to make the tension of the transmission element **310** change within a range of values, the driving position **510** where the power generating device **300** is mounted is preferred but not limited to be disposed between the first pivot point **416** and the second pivot point **426**. For example, 5 the driving position **510** is disposed on the connecting line of the first pivot point **416** and the second pivot point **426**, or the driving position **510** is disposed on the area between the first pivot point **416** and the second pivot point **426**.

In addition, as shown in **FIG. 5**, the wheel axle point **435** of the third suspension arm **430**, the front swing end **432** and the rear swing end **434** may be disposed such that their axial 10 directions **435a**, **432a** and **434a** are parallel to each other. In a related trial, such arrangement has the benefits of minimizing the distance **d** between the wheel axle point **435** of the third suspension arm **430** and the driving position **510**, and makes sure that the first swing point **436** where the front swing end **432** is pivoted to the first swing end **414** of the first suspension arm **410**, and the second swing point **438** where the rear swing end **434** is 15 pivoted to the second swing end **424** of the second suspension arm **420** are both with great structural strength.

With reference to **FIG. 1** again, although the embodiment merely discloses that the two ends **442** of the shock absorber component **440** are pivotally connected to the first fixed end **412** of the first suspension arm **410** and the second swing end **424** of the second suspension 20 arm **420** respectively. However, those of ordinary skill in the art can also appreciate that in the case that the two ends **442** of the shock absorber component **440** are pivotally connected

to the first swing end **414** of the first suspension arm **410** and the second fixed end **422** of the second suspension arm **420** respectively (see **FIG. 4**), the suspension structure **400** can still reach the preferable effects.

Further, with reference to **FIG. 5** again, the first pivot point **416** and the second pivot point **426** may be disposed such that their axial directions **416a** and **426a** are parallel or non-parallel. This also assists to make the wheel axle point **210** of the driving wheel **200** keep a constant distance **d** to the driving position **510** while the vehicle **500** driving, or make the distance **d** between the wheel axle point **210** of the driving wheel **200** and the driving position **510** merely change within a range of values while the vehicle **500** driving.

Although the vehicle **500** is illustrated as a two-wheel vehicle in the above-mentioned embodiments, the vehicle **500** may also be a three-wheel vehicle or a four-wheel vehicle. In addition, the power generating device **300** is preferred but not limited to a motor or an engine, and the transmission element **310** is preferred but not limited to a belt or a chain according to the present invention.

As mentioned above, the suspension structure **400** of the driving assembly **100** is adapted to effectively absorb the impact force generated by the environment and provide comfortable traveling quality for the goods and the passengers while the vehicle **500** driving. Furthermore, by the arrangement of the suspension structure **400**, the first suspension arm **410**, the second suspension arm **420**, the third suspension arm **430** and the shock absorber component **440**, the wheel axle point **210** of the driving wheel **200** is able to keep a constant distance to the driving position **510**, or make the distance between the wheel axle point **210** of

the driving wheel **200** and the driving position **510** merely change within a range of values. Therefore, the life of the transmission element **310** is extended, and the power conversion from the power generation device **300** to the driving wheel **200** is optimized.

The above disclosure is related to the detailed technical contents and inventive features
5 thereof. People skilled in this field may proceed with a variety of modifications and replacements based on the disclosures and suggestions of the invention as described without departing from the characteristics thereof. Nevertheless, although such modifications and replacements are not fully disclosed in the above descriptions, they have substantially been covered in the following claims as appended.

What is claimed is:

1. A suspension structure utilized in a driving assembly of a vehicle, and adapted to pivotally connected with a driving wheel of the driving assembly, wherein the driving wheel comprises a rotating axis, the suspension structure comprising:

a first suspension arm, having a first fixed end and a first swing end opposite to the first fixed end, wherein the first fixed end is pivoted to the vehicle at a first pivot point;

a second suspension arm, having a second fixed end and a second swing end opposite to the second fixed end, wherein the second fixed end is pivoted to the vehicle at a second pivot point;

a third suspension arm, having a front swing end and a rear swing end opposite to the front swing end, the front swing end being pivotally connected to the first swing end of the first suspension arm at a first swing point, and the rear swing end being pivotally connected to the second swing end of the second suspension arm at a second swing point; and

a shock absorber component, disposed between the first suspension arm and the second suspension arm and having two ends pivotally connected to the first suspension arm and the second suspension arm respectively.

2. The suspension structure of claim 1, wherein the arrangement between the first suspension arm and the second suspension arm is in a non-parallel relationship.

3. The suspension structure of claim 1, wherein the third suspension arm further comprises a wheel axle point disposed between the front swing end and the rear swing end, and the rotating axis of the driving wheel is disposed on the wheel axle point.

4. The suspension structure of claim 3, wherein the driving wheel is connected to a power generating device by a transmission element.

5. The suspension structure of claim 4, wherein the power generating device is disposed on a driving position of the vehicle.

6. The suspension structure of claim 5, wherein the wheel axle point keeps a constant distance to the driving position while the vehicle driving.

7. The suspension structure of claim 5, wherein the driving position is disposed between the first pivot point and the second pivot point.

8. The suspension structure of claim 1, wherein the two ends of the shock absorber component are pivotally connected to the first fixed end of the first suspension arm and the second swing end of the second suspension arm respectively.

9. The suspension structure of claim 1, wherein the two ends of the shock absorber component are pivotally connected to the first swing end of the first suspension arm and the second fixed end of the second suspension arm respectively.

10. The suspension structure of claim 1, wherein the vehicle is a two-wheel vehicle, a three-wheel vehicle or a four-wheel vehicle.

11. A driving assembly utilized in a vehicle, comprising:

a driving wheel, comprising a rotating axis;

a power generating device, disposed on a driving position of the vehicle and connected to the driving wheel by a transmission element for providing a power to the driving wheel; and

a suspension structure, adapted to pivotally connected with the driving wheel, comprising:

a first suspension arm, having a first fixed end and a first swing end opposite to the first fixed end, wherein the first fixed end is pivoted to the vehicle at a first pivot point;

a second suspension arm, having a second fixed end and a second swing end opposite to the second fixed end, wherein the second fixed end is pivoted to the vehicle at a second pivot point;

a third suspension arm, having a front swing end and a rear swing end opposite to the front swing end, the front swing end is pivotally connected to the first swing end of the first suspension arm at a first swing point and the rear swing end is pivotally connected to the second swing end of the second suspension arm at a second swing point; and

a shock absorber component, disposed between the first suspension arm and the second suspension arm and having two ends pivotally connected to the first suspension arm and the second suspension arm respectively.

12. The driving assembly of claim 1, wherein the third suspension arm further comprises a wheel axle point disposed between the front swing end and the rear swing end, and the rotating axis of the driving wheel is disposed on the wheel axle point.

13. The driving assembly of claim 12, wherein the wheel axle point keeps a constant distance to the driving position while the vehicle driving.

14. The driving assembly of claim 12, wherein the distance between the wheel axle point and the driving position changes within a range of values while the vehicle driving.

15. The driving assembly of claim 11, wherein the driving position is disposed between the first pivot point and the second pivot point.

16. The driving assembly of claim 11, wherein the two ends of the shock absorber component are pivotally connected to the first fixed end of the first suspension arm and the second swing end of the second suspension arm respectively.

17. The driving assembly of claim 11, wherein the two ends of the shock absorber component are pivotally connected to the first swing end of the first suspension arm and the

second fixed end of the second suspension arm respectively.

18. The driving assembly of claim 11, wherein the arrangement between the first suspension arm and the second suspension arm is in a non-parallel relationship.

19. The driving assembly of claim 11, wherein the vehicle is a two-wheel vehicle, a three-wheel vehicle or a four-wheel vehicle.

20. The driving assembly of claim 11, wherein the power generating device is a motor or an engine, and the transmission element is a belt or a chain.

100

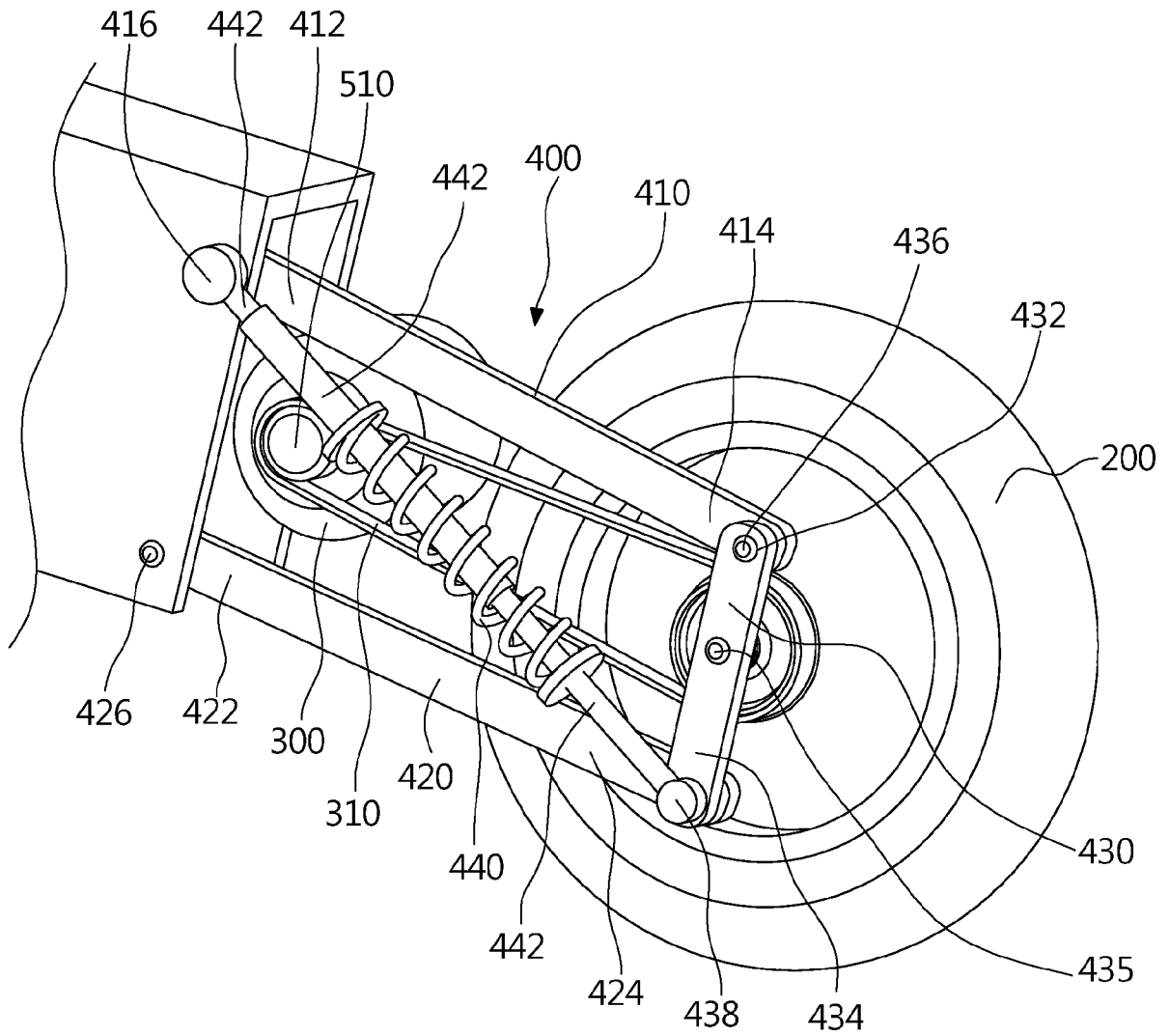


FIG.1

500

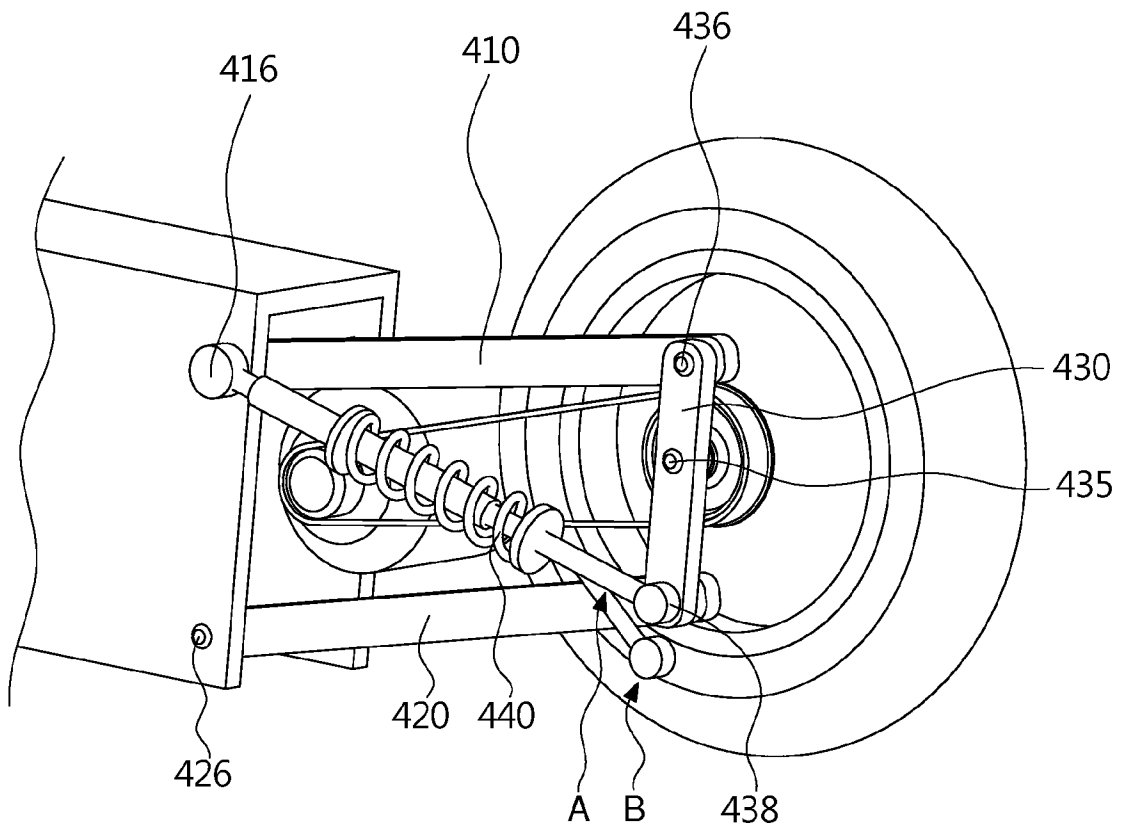


FIG.2

3/5

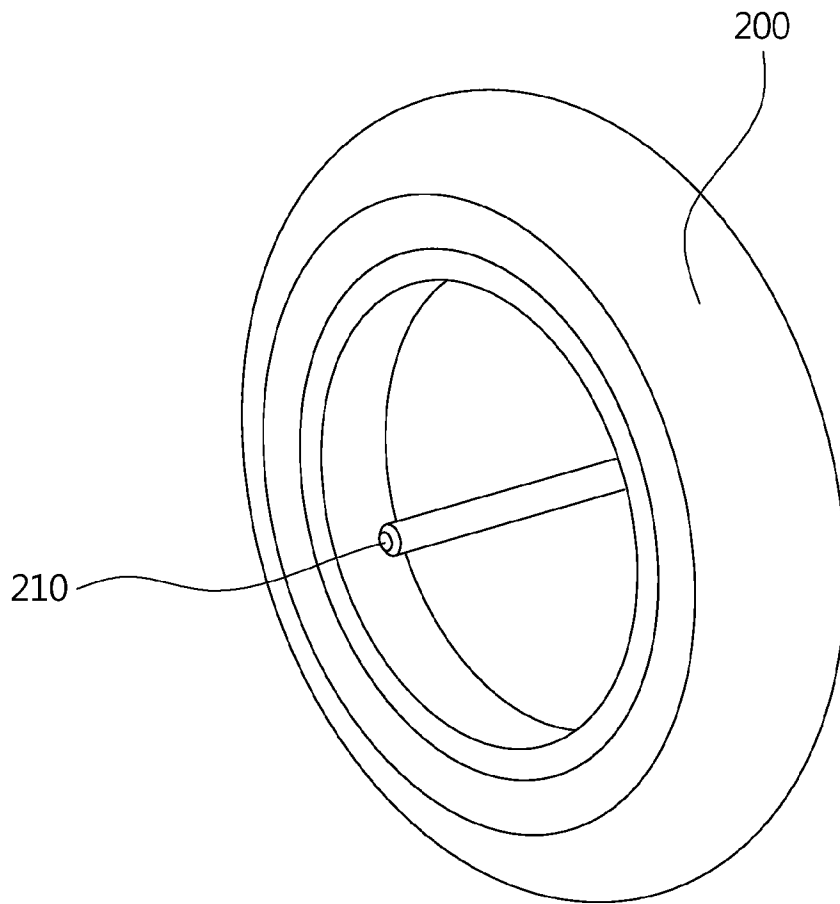


FIG.3

500

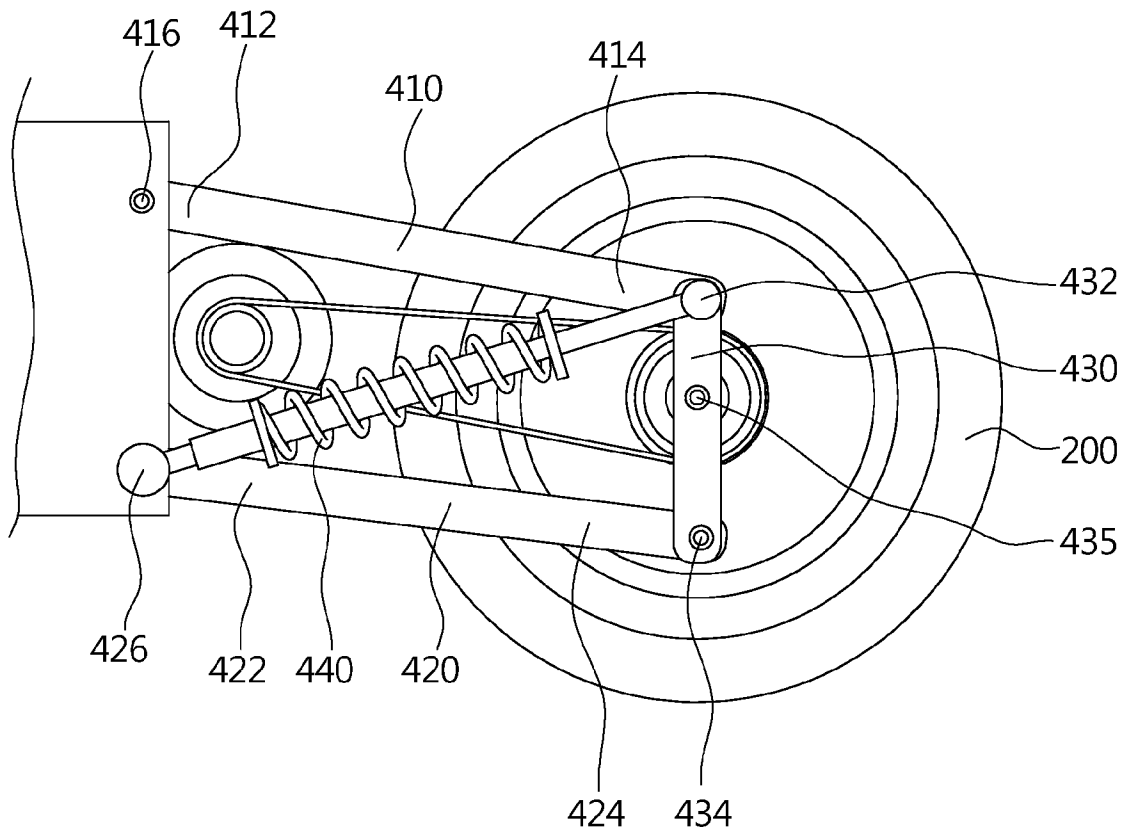


FIG.4

5/5

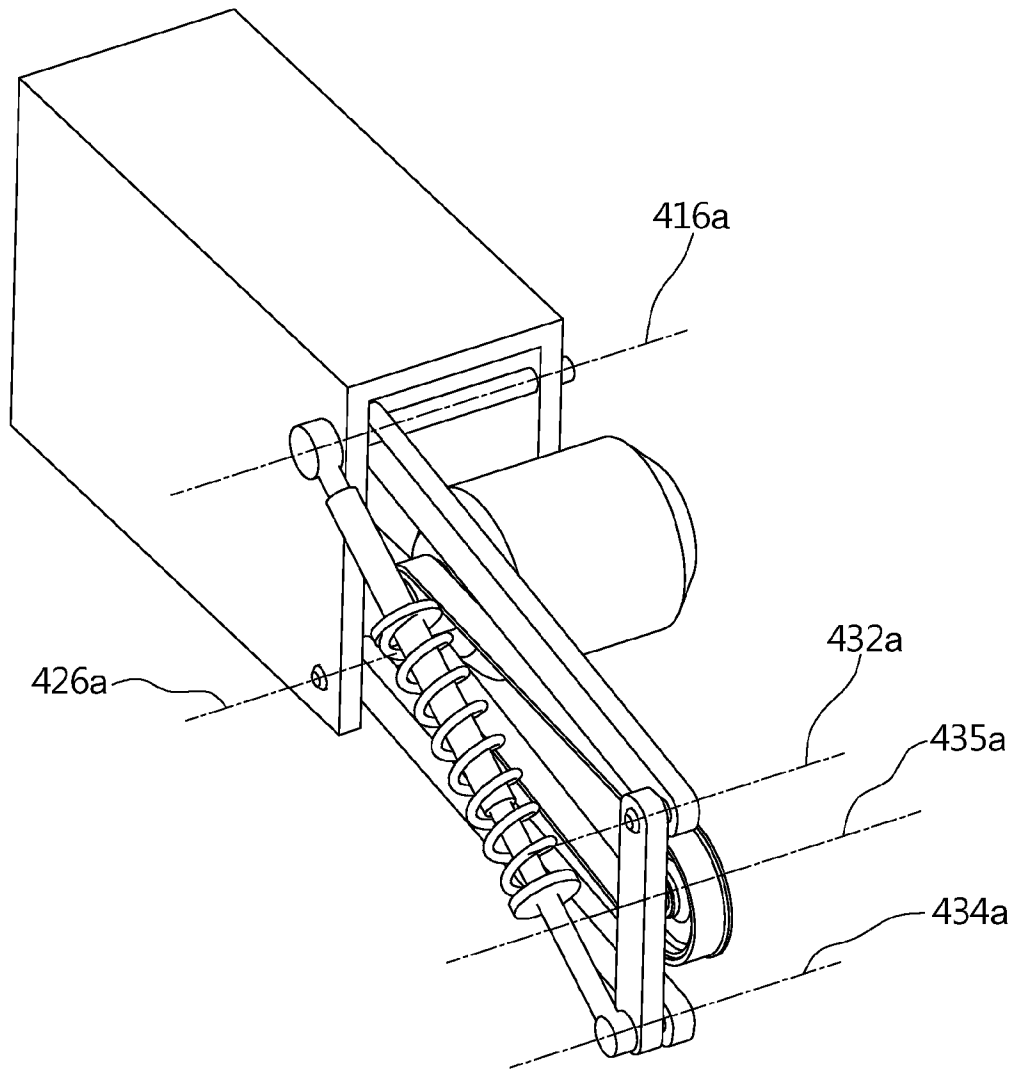


FIG.5

INTERNATIONAL SEARCH REPORT

International application No.
PCT/US2014/022828

A. CLASSIFICATION OF SUBJECT MATTER

IPC(8) - B60G 3/18 (2014.01)

USPC - 280/124.136

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

IPC(8) - B60G 3/18, 3/20, 5/04, 11/26, 11/58, 15/00, 15/06; B62K 25/22 (2014.01)

USPC - 280/5.5, 5.52, 89.12, 124.134, 124.135, 124.136, 124.141, 284

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

CPC - B60G 3/18, 3/20, 5/04, 11/26, 11/58, 15/00, 15/06; B62K 25/22 (2014.02)

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)

Orbit, Google Patents, Google

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	US 2,226,406 A (KROTZ) 24 December 1940 (24.12.1940) entire document	1-3, 8-10
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Y		4-7, 11-20
Y	US 7,690,661 B2 (TSURUTA et al) 06 April 2010 (06.04.2010) entire document	4-7, 11-20
A	US 2,621,919 A (UTZ) 16 December 1952 (16.12.1952) entire document	1-20
A	US 2,162,828 A (SLACK) 20 June 1939 (20.06.1939) entire document	1-20
A	US 6,746,032 B2 (SEKI) 08 June 2004 (08.06.2004) entire document	1-20

Further documents are listed in the continuation of Box C.

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Date of the actual completion of the international search
24 June 2014

Date of mailing of the international search report
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