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ELEVATOR AND ELEVATOR BRAKE

The present invention relates to an elevator as presented the
in the preamble of claim 1 and an elevator brake as presented
5 in the preamble of claim 4.

According to the regulations elevators must contain an
emergency brake, which is switched on in exceptional
situations, such as during an electricity power cut. The
10 normal holding brake of the elevator, for example, is used as
an emergency brake, which keeps the elevator car in its
position in normal situations when the car is at a landing.
Deceleration of the car in normal situations is usually
effected as motor braking. Connection of the emergency brake
15 when moving upwards in elevators without counterweight,
combined with the earth's gravity, can cause such large
deceleration of the elevator car that the safety of the
passengers can be endangered. Thus when traveling upwards the
emergency brake should not decelerate the elevator car of an
20 elevator without counterweight too effectively or not
necessarily at all.

The purpose of this invention is to eliminate the
aforementioned drawback and to achieve a simple and
25 operationally reliable elevator and elevator brake, which
solution does not cause unnecessary discomfort or dangerous
situations for the passengers. The elevator according to the
invention is characterized by what is disclosed in the
characterization part of claim 1 and the elevator brake
30 according to the invention is characterized by what is
disclosed in the characterization part of claim 4. Other
embodiments of the invention are characterized by what is
disclosed in the other claims.

35 Some inventive embodiments are also discussed in the
descriptive section of the present application. The inventive
content of the application can also be defined differently
than in the claims presented below. The inventive content may
also consist of several separate inventions, especially if the

invention is considered in the light of expressions or implicit sub-tasks or from the point of view of advantages or categories of advantages achieved. In this case, some of the attributes contained in the claims below may be superfluous
5 from the point of view of separate inventive concepts. Likewise the different details presented in connection with each embodiment of the invention can also be applied in other embodiments. Problems solved separately can be for example adjustment of the air gap of the brake or the guiding of
10 supporting forces to support in braking occurring in the upward or downward direction.

In the invention the braking exerted on the traction sheave of the brake is reduced in a situation where emergency braking
15 occurs, in which the elevator car is moving upwards, by allowing the brake shoe to follow the movement of the traction sheave for at least some distance. Preferably the normal force exerted on the brake wheel of the brake shoe at the same time is simultaneously allowed to be lighter.

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The advantage of the elevator according to the invention and its braking solution is that the upward movement of the elevator car in elevators without counterweight does not slow down too quickly in an emergency braking situation, in which
25 case no danger is caused to the passengers in the car by sudden stopping of the elevator car. A further advantage is the dependable and operationally reliable construction. Another advantage is that the solution according to the invention is suitable for use also in elevators with
30 counterweight, in which case slipping between the hoisting rope and the traction wheel and between the hoisting rope and the diverting pulleys caused by sudden deceleration can be prevented. This saves at least both the rope and the traction wheel and diverting pulleys.

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In the following, the invention will be described in more detail by the aid of some example of its embodiment with reference to the attached drawings, wherein

Fig. 1 presents a diagrammatic and simplified side view of a traction sheave elevator without counterweight applicable to the solution according to the invention,

Fig. 2 presents a side view of a sectioned part of the brake appliance of an elevator, in which one embodiment of the solution according to the invention is used and

Fig. 3 presents a side view of a sectioned part of a brake appliance of an elevator according to Fig. 2, in which the brake lightening function is switched on.

Fig. 1 presents a diagrammatic and simplified side view of one traction sheave elevator without counterweight applicable to the solution according to the invention, which comprises at least an elevator hoisting machine 5 with hoisting motor, a traction sheave 6, an elevator control system 8, as well as an elevator car 1 in a car sling 2 moving in an essentially vertical direction along guide rails 4, which is suspended on hoisting ropes 3. The first end of the hoisting ropes 3 is fastened to the top part of the car sling 2, from where it is led to pass around and over the traction sheave 6, and then under the diverting pulley 7 of the hoisting machine 5 and next over the traction sheave 6 again, from where it goes under the diverting pulley 9 fixed to the bottom of the elevator shaft, and then on to the bottom part of the car sling 2, to which the second end of the hoisting ropes 3 is fastened. The elevator receives its lifting force from the hoisting machine 5 as a result of the friction between the traction sheave 6 and the hoisting ropes 3. The roping suspension presented in Fig. 1 is a simplified roping construction for an elevator without counterweight. Often the hoisting rope 3 is led to pass via numerous diverting pulleys, so that the suspension ratio is of the magnitude desired. In addition the hoisting machine contains e.g. two brakes 10, which are positioned on opposite sides of the center axis. The

brake 10 is primarily a holding brake, which keeps the elevator car 1 in its position when the car is at a landing. Normally braking of the movement of the elevator car occurs as motor braking by means of the hoisting motor. The brake 10 is also used for emergency braking, in which case very great deceleration is achieved with the brake, especially in an elevator without counterweight, owing to the good friction between the hoisting ropes and the traction sheave 6. Deceleration that is too great is a problem, especially in elevators without counterweight when moving upwards, as mentioned above. For this reason the braking force of the brake 10 according to this invention is reduced by lightening the compression of the brake shoes on the brake wheel, if the brake 10 is switched on while the elevator car 1 is moving upwards.

Fig. 2 presents a side view of a sectioned part of the brake appliance of an elevator, in which one embodiment of the solution according to the invention is used. The brake is presented in a situation in which the brake is activated while the elevator is stationary, but the brake is in almost the same position also when emergency braking is effected when the elevator car is moving downwards.

The hoisting machine 5 comprises at least a fixed stator and a rotating rotor, along with which a brake wheel 11 provided with a braking surface 11a rotates. Additionally a traction sheave 6 is fixed to the rotor so that it rotates with the rotor. The brake 10 comprises at least a brake frame 13 fixed to the hoisting machine 5, containing at least one electromagnet 20 equipped with windings. A brake plate 15 fitted to move in relation to the frame is located on the brake frame 13, between the brake wheel 11 and the frame 13, and a brake shoe 12 provided with a wedge piece 16 is further located between the brake plate 15 and the brake wheel 11 such that the wedge piece 16 is between the brake plate 15 and the brake shoe 12. The wedge surface of the wedge piece 16 on the side of the brake plate 15 and correspondingly the

surface of the brake plate 15 on the side of the wedge piece together form a substantially straight first sliding surface 16a, which is inclined to an angle, which opens the brake wheel 11 in that direction of rotation in which the elevator
5 car 1 moves upwards. The aforementioned direction of rotation is marked in the figure with the arrow A. In addition the second sliding surface 25 between the wedge piece 16 and the brake shoe 12 as viewed from the side is curved such that the wedge piece 16 slides on the sliding surface 25 in relation
10 to the brake shoe 12, but the wedge piece 16 does not come out from between the brake plate 15 and the brake shoe 12. Preferably the wedge angle between the brake plate 15 and the wedge piece 16 is appreciably greater than the friction angle between them, and even better: in addition to or alongside a
15 sufficient wedge angle the ease of a change in the interpositioning of the brake plate 15, the wedge piece 16 and the brake shoe 12 is achieved with a selection of materials suitable for these pieces in which case the friction between the brake plate 15, the wedge piece 16 and the brake shoe 12
20 at their mutual sliding surfaces is slight.

The brake frame 13 contains a center hole in the center of the frame and extending through the frame, in which is a bore 24 extending through the frame on the side of the brake plate 15 and at least a second bore 22 with the same center, which
25 is open to the opposite side of the brake plate 15 of the frame 13. The bore 22 does not extend through the frame 13 and the diameter of the bore 22 is greater than the diameter of the bore 24 such that at the meeting point of the bores inside the frame is a ring-like collar 21. In addition a
30 presser pin 18 is fitted to pass through the bores 22, 24, of which the diameter of the part mainly situated in the bore 24 is greater than the diameter of the part mainly in the bore 22, such that there is a ring-like collar 19 on the presser pin 18 at the meeting point of the different diameters. The
35 length of the part of the presser pin 18 of greater diameter is selected to be such that it is essentially greater than the length of the smaller diameter bore 24 of the center hole. A disc spring pack 14 locked at its first end with a

locking element 23 is located around the presser pin 18 in the larger diameter bore 22, which is fitted to press the presser pin 18 at its second end via the collars 19 towards the brake shoe 12. The brake plate 15 contains a stop
5 surface, against which the presser pin 18 presses and causes movement of the brake plate towards the brake shoe 12 in the braking phase.

In a braking situation current is disconnected from the
10 electromagnets 20, whereupon the disc spring pack 14 presses the presser pin on the collar 19 towards the brake wheel 11. The presser pin 18 for its part presses the brake plate 15, and the brake shoe 12 via both the brake plate and the wedge piece 16, towards the brake wheel 11. The collars 21 in the
15 smaller diameter bore 24 of the frame prevent the disc spring pack 14 from pressing the presser pin 18 too long a distance towards the brake wheel 11. When the brake is not activated, the electromagnets 20 keep the brake shoe 12 detached from the braking surface 11a of the brake wheel 11. Fig. 2
20 presents only one brake 10, the other is situated for example on the opposite side of the hoisting machine 5. In this case the brake 10 is on the upper and lower part of the hoisting machine, while in the solution according to Fig. 1 it is on the sides of the hoisting machine.

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Fig. 3 presents a part of a brake appliance of an elevator according to the invention in a situation in which the brake lightening function is switched on, when the emergency braking occurs while the elevator car is moving upwards. The
30 idea of the invention is that when the elevator car 1 is moving upwards, the brake 10 does not substantially slow down the speed, even if the emergency braking is switched on e.g. in the event of an electricity power cut. Likewise, when the elevator car 1 is stationary or is moving downwards and the
35 brake 10 is connected, the brake 10 operates in the conventional manner of a brake - i.e. it keeps the car 1 in its position or slows down downward movement. This is

possible with the same brake due to the wedge solution of the invention.

When the elevator car 1 is moving upwards the brake wheel 11 rotates in the situation of the figure counterclockwise in the direction of the arrow A. When the brake 10 is switched on while driving upwards, the mutual shaping of the wedge piece 16 on the brake shoe 12, the brake plate 15 and the brake shoe 12 enable the brake shoe 12 to move along with the brake wheel 11 in the direction of rotation of the braking surface 11a owing to the friction between the shoe and the braking surface 11a, in other words in the direction of the arrow A. The wedge piece 16 then moves in relation to the sliding surfaces 16a and 25 and at the same time the brake shoe 12 moves away from the braking surface 11a. The brake shoe 12 does not however manage to move too long a distance, because the spring-action stop and return element 17 supported against the frame prevents movement of the brake shoe 12 after a pre-determined distance. When the brake shoe 12 has moved far enough to the side below the brake plate 15, the lowest springs of the disc spring pack 14 engage the collar 21 in the center hole of the frame 13, whereupon the presser pin 18 no longer presses the brake plate 15 closer to the brake wheel 11. In this case the aforementioned compression towards the brake wheel 11 is no longer exerted on the brake shoe 12 and the brake shoe 12 is able to move a little away from the braking surface 11a, whereupon there is no longer any substantial braking force and the elevator car stops softly under the influence of the earth's gravity.

When the movement of the elevator car 1 upwards has stopped due to the earth's gravity, the elevator car simultaneously tries to move immediately downwards. In this case the brake shoe 12 wedges back into its normal position due to the spring force of the stop and return element 17 and also due to the friction between the brake shoe 12 and the braking surface 11a, if the elevator car 1 has continued its downward movement after stopping. The brake 10 then stops the elevator car in its position and the brake thereafter operates as a normal

holding brake. By adjusting the spring force of the stop and return element it is also possible to set the force of braking occurring in the upward direction to at least some extent.

5 The arrester element 26, which is presented in the figure as an adjuster screw, determines together with the spring of the return element 17 the resting position of the brake shoe 12. The arrester element 26 also prevents, at least for its part, the brake shoe from following the movement of the brake wheel
10 when braking occurs in the downward direction. The arrester element 26 receives support for bearing braking in the downward direction from the brake plate 15 or possibly from another part of the brake. The adjuster screw functioning as the arrester element also determines within the scope of its
15 adjustment tolerance the height of the entity jointly formed by the brake shoe 12, the brake plate 15 and the wedge piece 16 in the direction of the brake stroke, in which case the air gap between the brake shoe 12 and the braking surface 11a is easily adjustable.

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It is obvious to the person skilled in the art that the invention is not limited solely to the example described above, but that it may be varied within the scope of the claims presented below. Thus, for example, the lightening of
25 the braking force in an emergency braking situation can also be implemented with separate device structures than that presented above. Instead of mechanical lightening of the braking force, the braking force can be lightened electrically with electromagnets or also hydraulically.

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It is also obvious to the person skilled in the art that the suspension ratio of the elevator can be greater than a 1:1 ratio. For example 2:1, 4:1, 8:1 ratios, and also other ratios, including uneven ratios, are easy to implement from
35 the standpoint of the invention.

It is further obvious to the person skilled in the art that the invention is suitable for use also, apart from in

elevators without counterweight, in e.g. those types of elevators with counterweight, in which the counterweight is at most the weight of the empty elevator car. In this case slipping between the hoisting rope and the traction wheel and
5 between the hoisting rope and the diverting pulleys caused by sudden deceleration can be prevented.

CLAIMS

1. Elevator, which contains at least a hoisting machine (5) equipped with a traction sheave (6), a brake (10) affecting
5 the rotational movement of the traction sheave and provided with at least one brake shoe (12) and a braking surface (11a), and hoisting ropes (3), which are fitted to move by means of the traction sheave (6) an elevator car (1) that moves along guide rails (4), **characterized** in that in an
10 emergency braking situation when the elevator car (1) is moving upwards the brake shoe (12) of the brake (10) is arranged to move in the direction of rotation of the braking surface (11a).

15 2. Elevator according to claim 1, **characterized** in that in an emergency braking situation when the elevator car (1) is moving upwards the movement of the brake shoe (12) in the direction of rotation of the braking surface (11a) is arranged to lighten the normal force between the brake shoe (12) and
20 the brake wheel (11).

3. Elevator according to claim 1 or 2, **characterized** in that the brake (10) of the elevator comprises at least a brake plate (15) fitted to move towards the braking surface (11a)
25 and a wedge piece (16) situated to move between the brake plate (15) and the brake shoe (12), and in that an essentially straight sliding surface (16a) in an inclined attitude is between the wedge piece (16) and the brake plate (15), along which sliding surface (16a) the wedge piece (16) is fitted to
30 slide essentially in the direction of rotation of the braking surface in an emergency braking situation.

4. Elevator according to claim 3, **characterized** in that the sliding surface (25) between the brake shoe (12) of the
35 elevator and the wedge piece (16) is curved.

5. Elevator brake, which comprises at least the frame (13) of the brake and a brake plate (15) fitted to move with respect

to the frame (13) as well as a brake shoe (12) provided with a braking surface (11a), fitted to move with respect to the frame (13), in which brake is at least one spring element (14) for activating the brake and at least one electromagnet (20) for opening the brake, **characterized** in that a wedge piece (16) is fitted to move between the brake plate (15) and the brake shoe (12).

6. Elevator brake according to claim 5, **characterized** in that the wedge piece (16) is fitted to move in relation to both the brake plate (15) and the brake shoe (12).

7. Elevator brake according to claim 5 or 6, **characterized** in that an essentially straight sliding surface (16a) in an inclined attitude is between the wedge piece (16) and the brake plate (15).

8. Elevator brake according to claim 5, 6 or 7, **characterized** in that the sliding surface (16a) is inclined at an angle which opens the brake wheel (11) in the direction of rotation (A), in which the elevator car moves upwards.

9. Elevator brake according to any of claims 5-8 above, **characterized** in that between the wedge piece (16) and the brake shoe (12) is an essentially curved sliding surface (25).

10. Elevator brake according to any of claims 5-9 above, **characterized** in that the brake (10) contains a spring-action stop and return element (17), which is fitted to stop the movement of the brake shoe (12) after a pre-determined distance in the direction of rotation (A) of the brake wheel (11) in which the elevator car moves upwards and to return the brake shoe (12) to its normal position.

11. Elevator brake according to any of claims 5-10 above, **characterized** in that the brake (10) contains an arrester element (26) transmitting the braking forces of the elevator in the downward direction between the brake shoe (12) and the

brake plate (15).

12. Elevator brake according to any of claims 5-11 above, **characterized** in that the force of braking occurring in the 5 brake (10) in the upward direction can be set.

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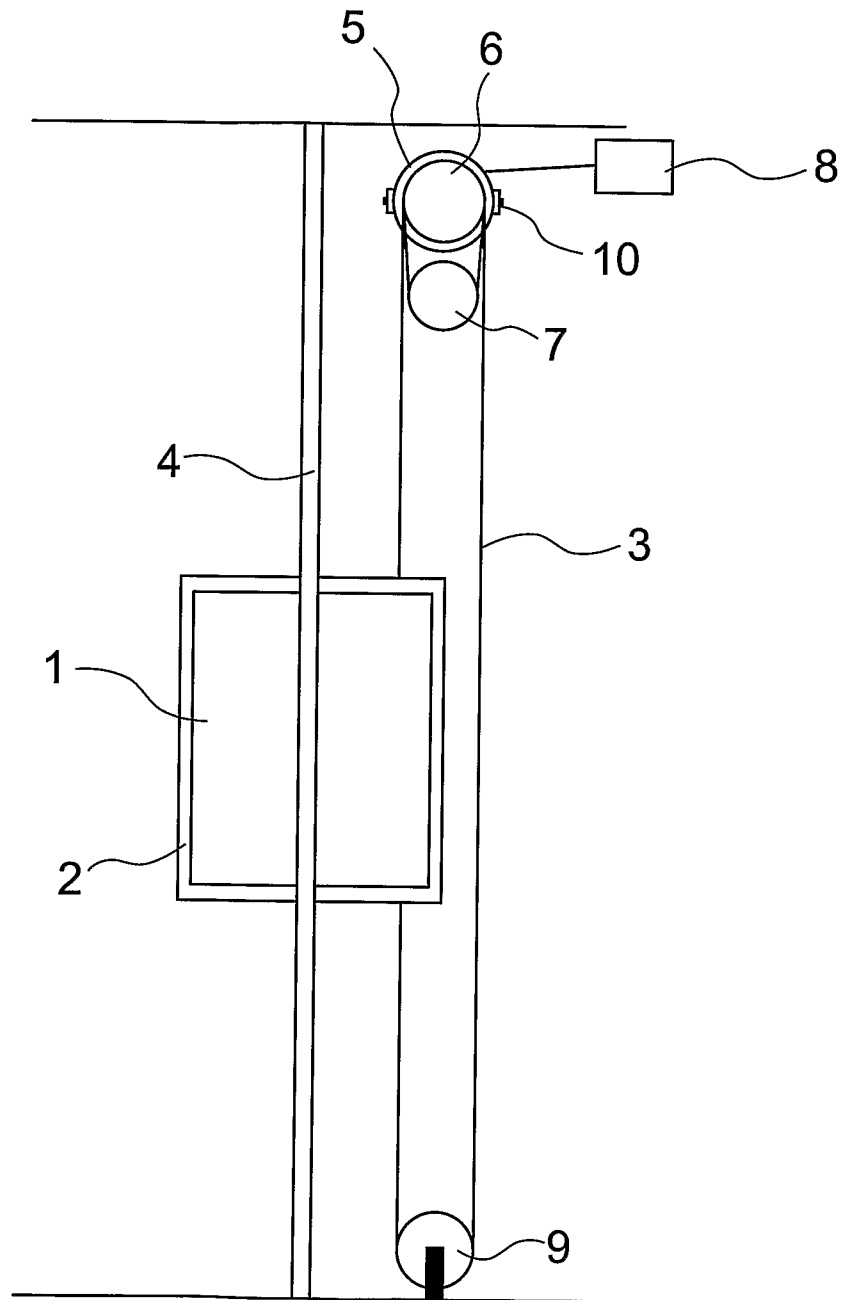


Fig. 1

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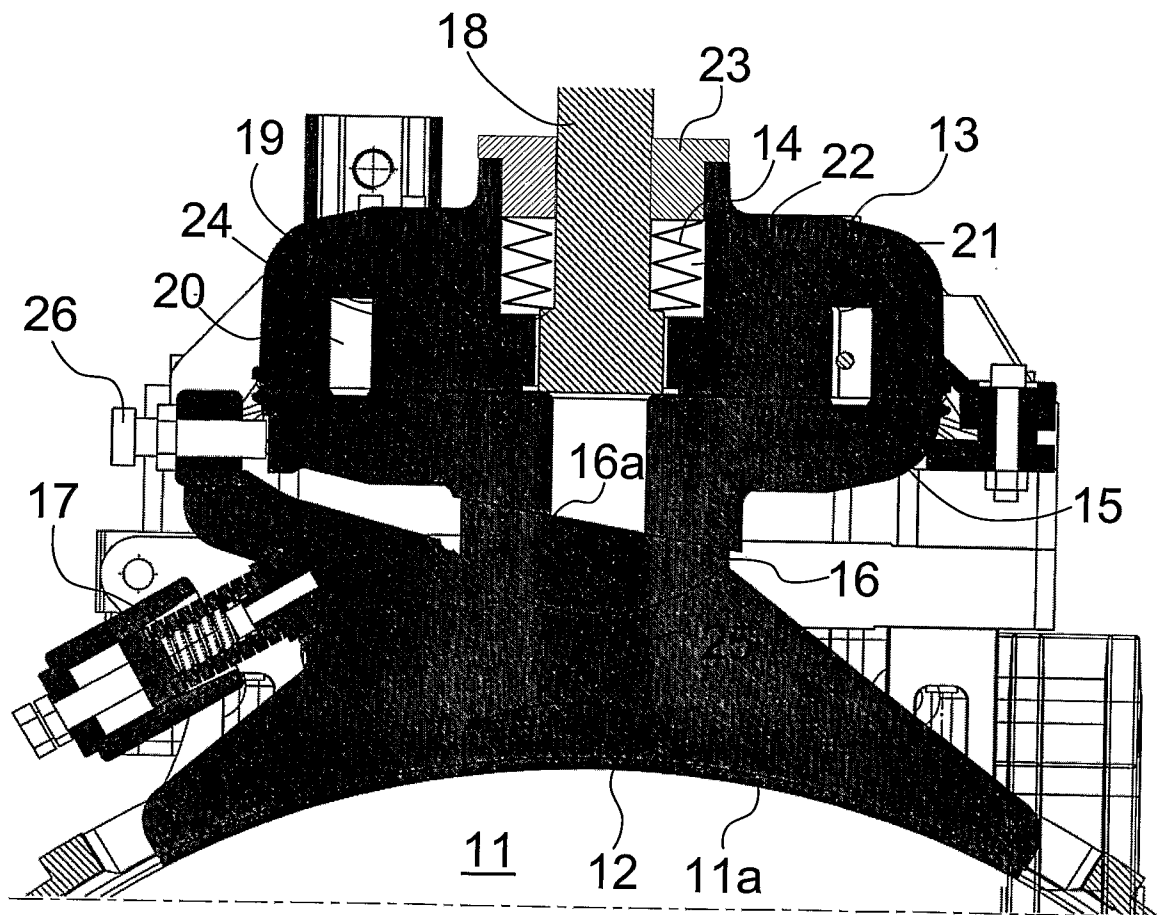


Fig. 2

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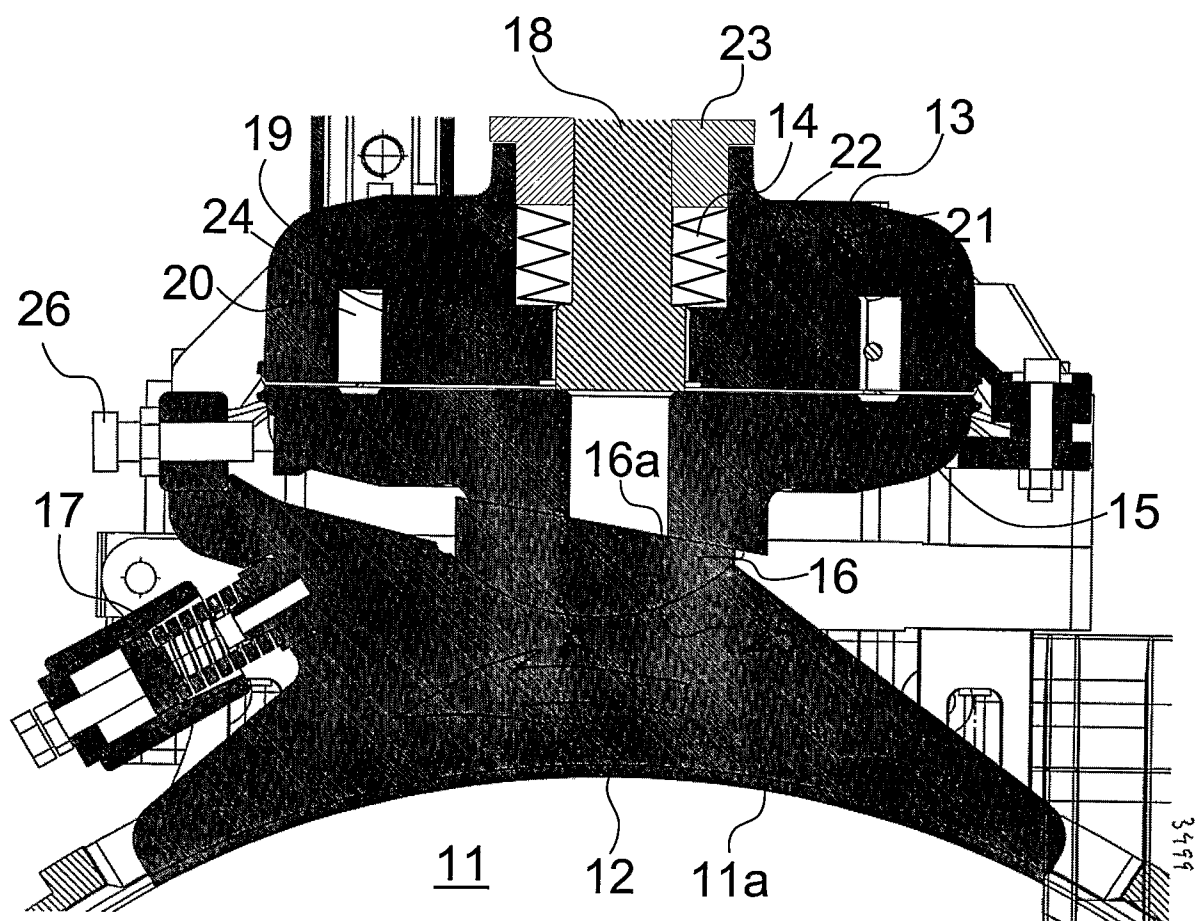


Fig. 3

INTERNATIONAL SEARCH REPORT

International application No

PCT/FI2007/000008

A. CLASSIFICATION OF SUBJECT MATTER

INV. B66D5/08

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

B66D

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

EPO-Internal

C. DOCUMENTS CONSIDERED TO BE RELEVANT

| Category* | Citation of document, with indication, where appropriate, of the relevant passages | Relevant to claim No. |
|-----------|---|-----------------------|
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| X | DE 22 64 615 A1 (GRUEMER EISENWERK) 12 September 1974 (1974-09-12) abstract claims 1-8 figures 1-13 | 1 |
| X | SU 1 252 298 A1 (VOROSHILOVGRAD MASHINOSTR [SU]) 23 August 1986 (1986-08-23) abstract figure 1 | 1 |
| | ----- -/-- | |

☒ Further documents are listed in the continuation of Box C.☒ See patent family annex.

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Name and mailing address of the ISA/

European Patent Office, P.B. 5818 Patentlaan 2
NL - 2280 HV Rijswijk
Tel. (+31-70) 340-2040, Tx. 31 651 epo nl,
Fax: (+31-70) 340-3016

Authorized officer

Oosterom, Marcel

INTERNATIONAL SEARCH REPORT

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C(Continuation). DOCUMENTS CONSIDERED TO BE RELEVANT

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INTERNATIONAL SEARCH REPORT

Information on patent family members

International application No

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