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(54) **Gyratory crusher outer crushing shell**

(57) A gyratory crusher outer crushing shell (106). The outer shell comprises three regions along its axial length including: an inlet region (121) that tapers radially inward from an uppermost first end (124); a crushing region (123) that extends radially inward from a second lowermost end (125) and; a radially innermost shoulder

region (122) that is positioned axially between the inlet and crushing regions. An angle of inclination (a,b) of a radially inward facing surface at the inlet and shoulder regions and the axial length (C) of the crushing surface are designed to optimise crushing capacity in addition to maximising reduction.

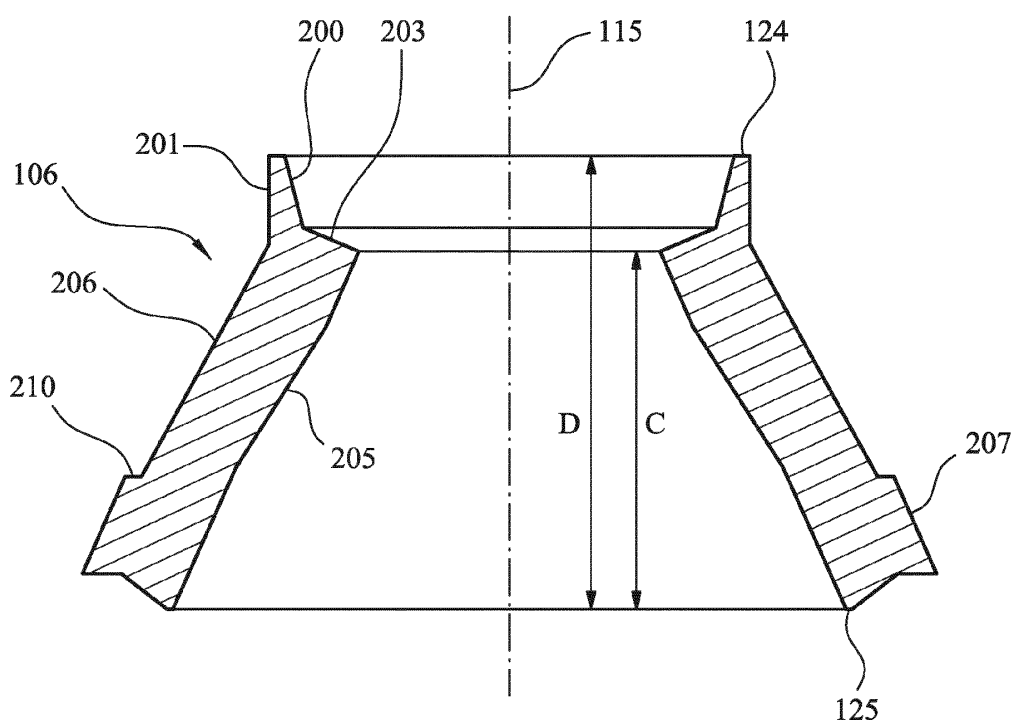


FIG. 3

Description

Field of invention

[0001] The present invention relates to a gyratory crusher outer crushing shell and in particular, although not exclusively, to a crushing shell having a radially inward projecting shoulder positioned axially intermediate between an upper inlet region and a lower crushing region, the inlet, shoulder and crushing region being optimised to increase the capacity and reduction effect of the crusher.

Background art

[0002] Gyratory crushers are used for crushing ore, mineral and rock material to smaller sizes. The crusher comprises a crushing head mounted upon an elongate main shaft. A first crushing shell (typically referred to as a mantle) is mounted on the crushing head and a second crushing shell (typically referred to as a concave) is mounted on a frame such that the first and second crushing shells define together a crushing chamber through which the material to be crushed is passed. A driving device positioned at a lower region of the main shaft is configured to rotate an eccentric assembly positioned about the shaft to cause the crushing head to perform a gyratory pendulum movement and crush the material introduced in the crushing chamber. Example gyratory crushers are described in WO 2004/110626; WO 2008/140375, WO 2010/123431, US 2009/0008489, GB 1570015, US 6,536,693, JP 2004-136252, US 1,791,584 and WO 2012/005651.

[0003] Gyratory crushers (encompassing cone crushers) are typically designed to maximise crushing efficiency that represents a compromise between crushing capacity (the throughput of material to be crushed) and crushing reduction (the breakdown of material to smaller sizes). This is particularly true for heavy-duty primary crushers designed for mining applications. The capacity and reduction may be adjusted by a variety of factors including in particular size of the crushing chamber, the eccentric mounting of the main shaft and the shape, configuration and setting of the opposed crushing shells.

[0004] For example, the design of the outer crushing shell has a significant effect on the capacity and reduction of the crusher. In particular, an outer crushing shell with an inner facing contact surface that tapers inwardly towards the mantle acts to accelerate the through-flow of material. However, conventional designs of this type fall short of optimising capacity whilst increasing reduction and there is therefore a need for an improved outer crushing shell with improved performance.

Summary of the Invention

[0005] It is an objective of the present invention to provide an outer crushing shell that is optimised to control

the throughput capacity and reduction of the crusher. It is a further objective to limit the throughput capacity in favour of reduction and to maximise the total net capacity for a specific application and type of crushable material.

[0006] The objectives are achieved, in part, by providing an outer crushing shell that is designed to decrease the throughput capacity via a shelf or shoulder region that restricts the flow of material through the crushing chamber in the gap between the opposed crushing shells. The creation of the shelf region is further advantageous to reduce the axial length of the shell which in turn decreases the available crushing surface area that is orientated to be radially inward facing towards the inner crushing shell. Advantageously, it has been found that restricting the capacity and crushing force area acts to increase the pressure in the crushing chamber in the gap region to increase the reduction effect.

[0007] In particular, the inventors have identified how variations of various physical parameters of the crushing shell influence capacity and reduction to enable optimisation of the geometry of the shell. The present crushing shell may be considered to comprise three regions spatially positioned in the axial direction between a shell uppermost end and a lowermost end. In particular, the present shell comprises an inlet region extending axially downward from the uppermost end, a crushing region extending axially upward from the lowermost end and a shoulder region positioned axially between the inlet and crushing regions. The inventors have observed that the following parameters influence the capacity and reduction of the crusher:

1. an angle of inclination of a radially inward facing surface at the inlet region;
2. an angle of inclination of a radially inward facing surface at the shoulder region;
3. a wall thickness at the shoulder region between a radially inward facing surface and a radially outward facing surface; and
4. an axial length of the crushing region relative to an overall axial length of the shell between its upper and lower ends.

[0008] According to a first aspect of the present invention there is provided a gyratory crusher outer crushing shell comprising a main body mountable within a region of a topshell frame of a gyratory crusher, the main body extending around a central longitudinal axis the main body having a mount surface being outward facing relative to the axis for positioning against at least a part of the topshell frame and a contact surface being inward facing relative to the axis to contact material to be crushed, at least one wall defined by and extending between the mount surface and the contact surface, the wall having a first upper axial end and a second lower axial end; an orientation of the contact surface extending from the first end being inclined so as to project radially inward towards the axis in the axially downward direction

to define an inlet region characterised in that an axially lowermost part of the inlet region is terminated by a shoulder region, a contact surface at the shoulder region being inclined so as to project radially inward towards the axis from the contact surface of the inlet region in an axially downward direction; wherein an angle of inclination of the contact surface of the inlet region relative to the axis is less than an angle of inclination of the contact surface of the shoulder region relative to the axis.

[0009] Optionally, the angle of inclination of the contact surface of the inlet region is in the range 1 to 40° relative to the axis. Preferably, the angle of inclination of the contact surface of the inlet region is in the range 4 to 12° relative to the axis.

[0010] Optionally, the angle of inclination of the contact surface of the shoulder region is in the range 45 to 90° relative to the axis. Preferably, the angle of inclination of the contact surface of the shoulder region is in the range 65 to 75° relative to the axis.

[0011] Optionally, an angle of inclination of the contact surface of the shoulder region is three to fifteen times greater than the angle of inclination of the contact surface of the inlet region relative to the axis. Preferably, the inlet region extends directly from the first upper axial end in the axial direction and the shoulder region extends directly from an axially lowermost part of the inlet region in the axial direction such that the contact surface comprises two surface regions of different inclination in the axial direction over the inlet region and the shoulder region from the first upper axial end.

[0012] Optionally, the contact surface from an axially lowermost part of the shoulder region to the second lower axial end defines a crushing face and comprises an axial length in the range of 40 to 85% of a total axial length of the main body from the first lower axial end to second lower axial end. Preferably the crushing face is orientated to be declined to project radially outward relative to the axis in a downward direction from the shoulder region to the second lower axial end.

[0013] A distance by which the contact surface at the shoulder region projects radially inward from a radially innermost region of the contact surface of the inlet region is optionally 5% to 90% and preferably 20% to 80%, 30% to 70%, 40% to 70%, 40% to 60%, 50% to 60% of a total radial thickness of the wall between the radially innermost shoulder part and the mount surface.

[0014] Optionally, a radially innermost part of the shoulder region is positioned in an upper 45%, 50% or 60% of an axial length of the main body closest to the first end and preferably in the range 5% to 30% of an axial length of the main body closest to the first end or 5% to 45%, 5% to 50% or 5% to 60%.

[0015] Optionally, a radially innermost part of the shoulder region is positioned at a region in the range 20 to 60% and preferably 20 to 45% of an axial length of the main body from the first end.

[0016] Preferably, the shell comprises one inlet region and one shoulder region such that the shell comprises

two inclined contact surfaces relative to axis and one declined contact surface relative to axis.

[0017] According to a second aspect of the present invention there is provided a gyratory crusher comprising a crushing shell as described herein.

[0018] Within the specification reference to a gyratory crusher encompasses primary, secondary and tertiary crushers in addition to encompassing cone crushers.

10 Brief description of drawings

[0019] A specific implementation of the present invention will now be described, by way of example only, and with reference to the accompanying drawings in which:

Figure 1 is a cross sectional elevation view of a gyratory crusher comprising an outer crushing shell (concave) and an inner crusher shell (mantle) according to a specific implementation of the present invention;

Figure 2 is magnified view of the region of the crusher of figure 1 illustrating the outer and inner crushing shells;

Figure 3 is a cross sectional elevation view of the outer crushing shell of figure 2;

Figure 4 is a magnified cross sectional elevation view of the upper region of the crushing shell of figure 3.

Detailed description of preferred embodiment of the invention

[0020] Referring to figure 1, a crusher comprises a frame 100 having an upper frame 101 and a lower frame 102. A crushing head 103 is mounted upon an elongate shaft 107. A first (inner) crushing shell 105 is fixably mounted on crushing head 103 and a second (outer) crushing shell 106 is fixably mounted at upper frame 101. A crushing zone 104 is formed between the opposed crushing shells 105, 106. A discharge zone 109 is positioned immediately below crushing zone 104 and is defined, in part, by lower frame 102.

[0021] A drive (not shown) is coupled to main shaft 107 via a drive shaft 108 and suitable gearing 116 so as to rotate shaft 107 eccentrically about longitudinal axis 115 and to cause head 103 and mantle 105 to perform a gyratory pendulum movement and crush material introduced into crushing chamber 104. An upper end region of shaft 107 is maintained in an axially rotatable position by a top-end bearing assembly 112 positioned intermediate between main shaft 107 and a central boss 117. Similarly, a bottom end 118 of shaft 107 is supported by a bottom-end bearing assembly 119.

[0022] Upper frame 101 is divided into a topshell 111, mounted upon lower frame 102 (alternatively termed a bottom shell), and a spider assembly 114 that extends

from topshell 111 and represents an upper portion of the crusher. The spider 114 comprises two diametrically opposed arms 110 that extend radially outward from central boss 117 positioned on longitudinal axis 115. Arms 110 are attached to an upper region of topshell 111 via an intermediate annular flange (or rim) 113 that is centred on axis 115. Typically, arms 110 and topshell 111 form a unitary structure and are formed integrally.

[0023] In the present example embodiment, the alignment of outer crushing shell 106 at topshell 111 is achieved by an intermediate spacer ring 120 that extends circumferentially around axis 115 and is positioned axially intermediate between spider 114 and topshell 111. Accordingly, an axially uppermost first end 124 of outer shell 106 is positioned radially inward within the circumference of spacer ring 120. An axially lowermost second end 125 of shell 106 is positioned just below a lowermost part of topshell 111 and approximately at the junction between bottom shell 102 and topshell 111.

[0024] Outer shell 106 principally comprises three regions in the axial direction: an uppermost inlet region 121 extending from first end 124; a crushing region 123 extending from second end 125 and a shoulder region 122 positioned axially intermediate between inlet region 121 and crushing region 123.

[0025] Referring to figure 2, inlet region 121 comprises a radially outward facing mount surface 201 that is aligned substantially parallel with axis 115. An opposed radially inward facing contact surface 200 is inclined radially inward from first end 124 such that a wall thickness of shell 106 at inlet region 121 increases uniformly from first end 124 to an axially lowermost base region 401 as shown in figure 4. The base region 401 of inlet region 121 terminates at shoulder region 122. Shoulder region 122 comprises a corresponding inward facing contact surface 203 that projects radially inward from inlet contact surface 200 to define a shelf 204 that represents a radially innermost region of shell 104. Crushing region 123 extends immediately below shoulder region 122 and also comprises inward facing contact surface 205 and an opposed outward facing mount surface 206. Contact surface 205 is orientated to be declined and projects away from axis 115 and towards topshell 111. An axially lowermost part 209 of crushing region 123 comprises a radially outward facing mount surface 207 configured for close mating contact against a radially inward facing surface 208 of a lower region of topshell 111 such that shell 106 is mounted against topshell 111 via contact between opposed surfaces 207, 208.

[0026] Referring to figures 3 and 4, a wall thickness of shell 106 increases from uppermost first end 124 over the axial length of inlet region 121 due to the inclined (or radially inward tapering) contact surface 200. The shell wall thickness increases further at shoulder region 122 via radially inward tapering contact surface 203. The wall thickness of shell 106 is then approximately uniform along the axial length of crushing region 123 until lowermost region 209 where the wall thickness projects radially

outward to create a mounting flange 210 for contact and mounting against topshell 111.

[0027] As will be appreciated, shell 106 extends circumferentially around axis 115. With regard to the outward appearance defined by respective mount surfaces 201, 206 and 207, inlet region 121 is substantially cylindrical and the shoulder region 122 and crushing region 123 are generally frusto-conical shaped.

[0028] As illustrated, shelf 204 is positioned at an axially uppermost part of shell 106 and, in particular, in the top 25% region closest to first end 124 referring to relative axial lengths C and D (where C is the distance between shelf 204 and second lowermost end 125 and D is the distance axially between first uppermost end 124 and second end 125).

[0029] Referring to figure 4, an angle of inclination a of contact surface 200 is approximately 10° from central axis 115 and an angle of inclination b of contact surface 203 is approximately 70° from central axis 115. As illustrated, both contact surfaces 200, 203 are substantially linear and extend circumferentially around axis 115. The junction between surfaces 200, 203 comprises a slight curvature. Distance F represents the maximum wall thickness of shell 106 at inlet region 121. Distance F is defined as the distance between outward facing mount surface 201 and radially inward facing contact surface 200 at the inlet base region 401 representing the point of intersection of respective contact surfaces 200, 203. Radial distance E is defined as the distance between intersection point 400 and the radially innermost point 204 of the shoulder region 122. A ratio of E to F according to the specific implementation is 1:0.8. That is, the distance E is approximately 55% of the total wall thickness (E+F) between the mount surface 201 and the radially innermost point of the shoulder region 204.

[0030] Advantageously, the combined and respective inclination of surfaces 200 and 203 via angles a and b serve to accelerate the throughput as material falls through inlet region 121 and is directed radially inward over shelf 124. However, increasing the radial length E of shelf 204 decreases the crushing capacity. The present configuration as illustrated in figures 1 to 4 is therefore optimised to control the capacity of the crusher and achieve a predetermined level specific to a particular application. Additionally, incorporating inlet region 121 and shoulder region 122 decreases the axial length of crushing surface 205 from length D to length C. The surface area of crushing surface 205 (that is approximately frusto-conical shaped) is therefore reduced which acts to increase the pressure in crushing region 104 where the crushing forces are applied during operation. This in turn increases the reduction effect of the crusher. The inventors have observed that the present relative configurations of inlet region 121; shoulder region 122 and crushing region 123 with regard to radial wall thicknesses, contact surface angles and axial lengths provides an optimised material throughput capacity and reduction and hence performance of the crusher. In particular, the

following four parameters, have been found to influence the performance of the shell 106 with regard to throughput capacity and reduction: i) angle *a* of contact surface 200; ii) angle *b* of contact surface 203; iii) a radial distance *E* of shelf 204 and; iv) an axial length *C* of crushing surface 205.

[0031] In particular the angle *a* of contact surface 200 relative to angle *b* of contact surface 203 defines the inlet 121 and shoulder 122 regions with these regions being significant to control capacity.

Claims

1. A gyratory crusher outer crushing shell (106) comprising:

a main body mountable within a region of a top-shell frame (111) of a gyratory crusher, the main body extending around a central longitudinal axis (115);

the main body having a mount surface (201, 206, 207) being outward facing relative to the axis (115) for positioning against at least a part of the topshell frame (111) and a contact surface (200, 203, 205) being inward facing relative to the axis (115) to contact material to be crushed, at least one wall defined by and extending between the mount surface (201, 206, 207) and the contact surface (200, 203, 205), the wall having a first upper axial end (124) and a second lower axial end (125);

an orientation of the contact surface (200) extending from the first end (124) being inclined so as to project radially inward towards the axis (115) in the axially downward direction to define an inlet region (121);

characterised in that:

an axially lowermost part (401) of the inlet region (121) is terminated by a shoulder region (122), a contact surface (203) at the shoulder region (122) being inclined so as to project radially inward towards the axis (115) from the contact surface (200) of the inlet region (121) in an axially downward direction;

wherein an angle of inclination (*a*) of the contact surface (200) of the inlet region (121) relative to the axis (115) is less than an angle of inclination (*b*) of the contact surface (203) of the shoulder region (122) relative to the axis.

2. The shell has claimed in claim 1 wherein the angle of inclination (*a*) of the contact surface (200) of the inlet region (121) is in the range 1 to 40° relative to the axis.

3. The shell has claimed in claim 1 wherein the angle of inclination (*a*) of the contact surface (200) of the inlet region (121) is in the range 4 to 12° relative to the axis.

4. The shell has claimed in claim 1 wherein the angle of inclination (*b*) of the contact surface (203) of the shoulder region (122) is in the range 45 to 90° relative to the axis.

5. The shell has claimed in claim 1 wherein the angle of inclination (*b*) of the contact surface (203) of the shoulder region (122) is in the range 65 to 75° relative to the axis.

6. The shell as claimed in any preceding claim wherein an angle of inclination (*b*) of the contact surface (203) of the shoulder region (122) is three to fifteen times greater than the angle of inclination (*a*) of the contact surface (200) of the inlet region (122) relative to the axis (115).

7. The shell as claimed in any preceding claim wherein the inlet region (121) extends directly from the first upper axial end (124) in the axial direction and the shoulder region (122) extends directly from an axially lowermost part of the inlet region (121) in the axial direction such that the contact surface comprises two surface regions of different inclination in the axial direction over the inlet region and the shoulder region from the first upper axial end (124).

8. The shell as claimed in any preceding claim wherein the contact surface (205) from an axially lowermost part of the shoulder region (122) to the second lower axial end (125) defines a crushing face and comprises an axial length (*C*) in the range of 40 to 85% of a total axial length (*D*) of the main body from the first upper axial end (124) to the second lower axial end (125).

9. The shell as claimed in claim 8 wherein the crushing face is orientated to be declined to project radially outward relative to the axis (115) in a downward direction from the shoulder region (122) to the second lower axial end (125).

10. The shell as claimed in any preceding claim wherein a distance (*E*) by which the contact surface (203) at the shoulder region (122) projects radially inward from a radially innermost region (400) of the contact surface (200) of the inlet region (121) is 5% to 90% of a total radial thickness of the wall between the radially innermost shoulder part (204) and the mount surface (201, 206).

11. The shell as claimed in any preceding claim wherein a ratio of a distance (*E*) by which the contact surface

(203) at the shoulder region (122) projects radially inward from a radially innermost region (400) of the contact surface (200) of the inlet region (121) is 40% to 70% of a total radial thickness of the wall between the radially innermost shoulder part (204) and the mount surface (201, 206). 5

12. The shell as claimed in any preceding claim wherein a radially innermost part (204) of the shoulder region (122) is positioned in an upper 60% of an axial length (D) of the main body closest to the first end (124). 10
13. The shell as claimed in any preceding claim wherein a radially innermost part (204) of the shoulder region (122) is positioned at a region in the range 20 to 45% of an axial length (D) of the main body from the first end (124). 15
14. The shell as claimed in any preceding claim comprising one inlet region (121) and one shoulder region (122) such that the shell (106) comprises two inclined contact surfaces (200, 203) relative to axis (115) and one declined contact surface (205) relative to axis (115). 20
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15. A gyratory crusher comprising a crushing shell (106) as claimed in any preceding claim. 30
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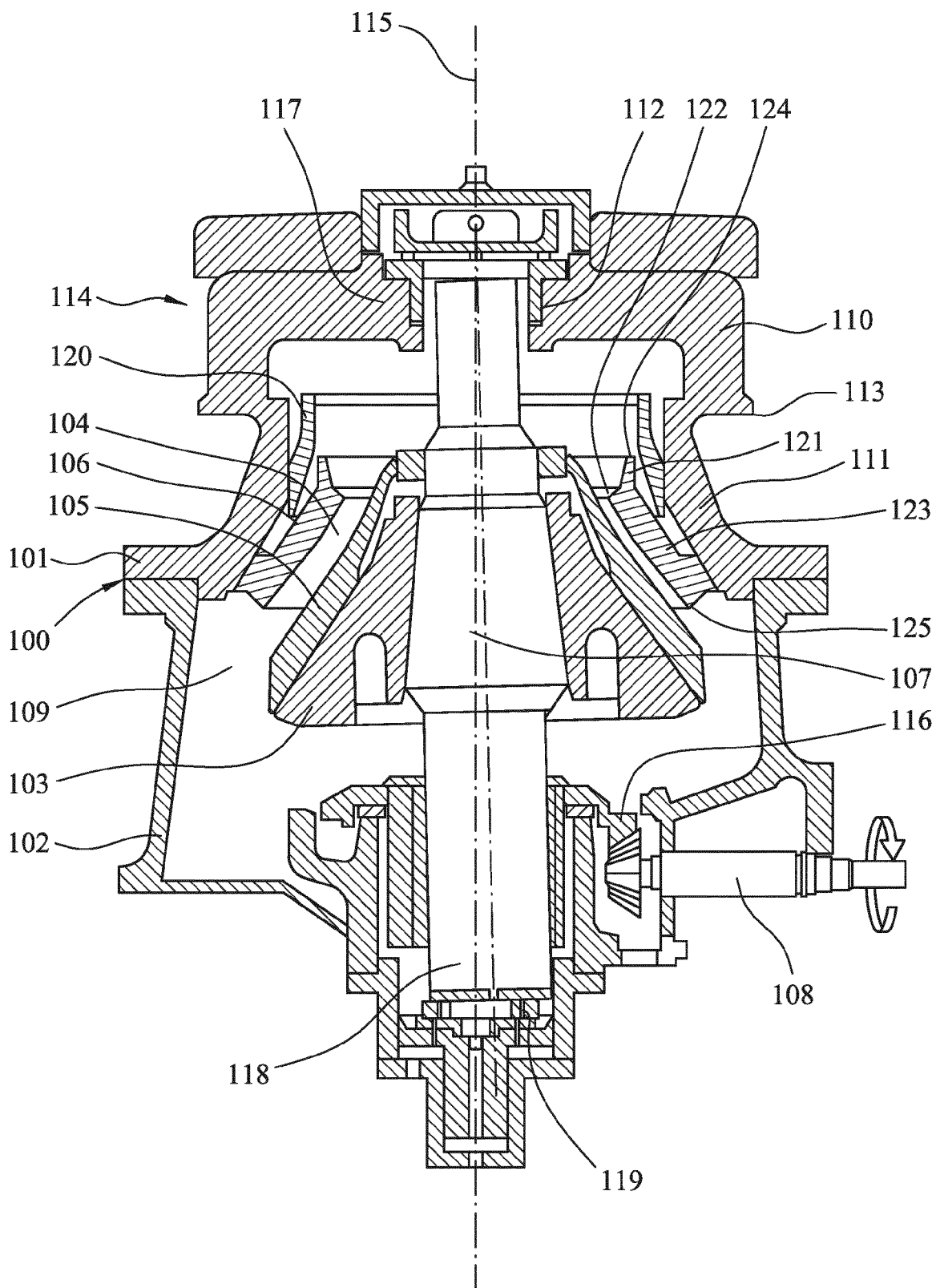


FIG. 1

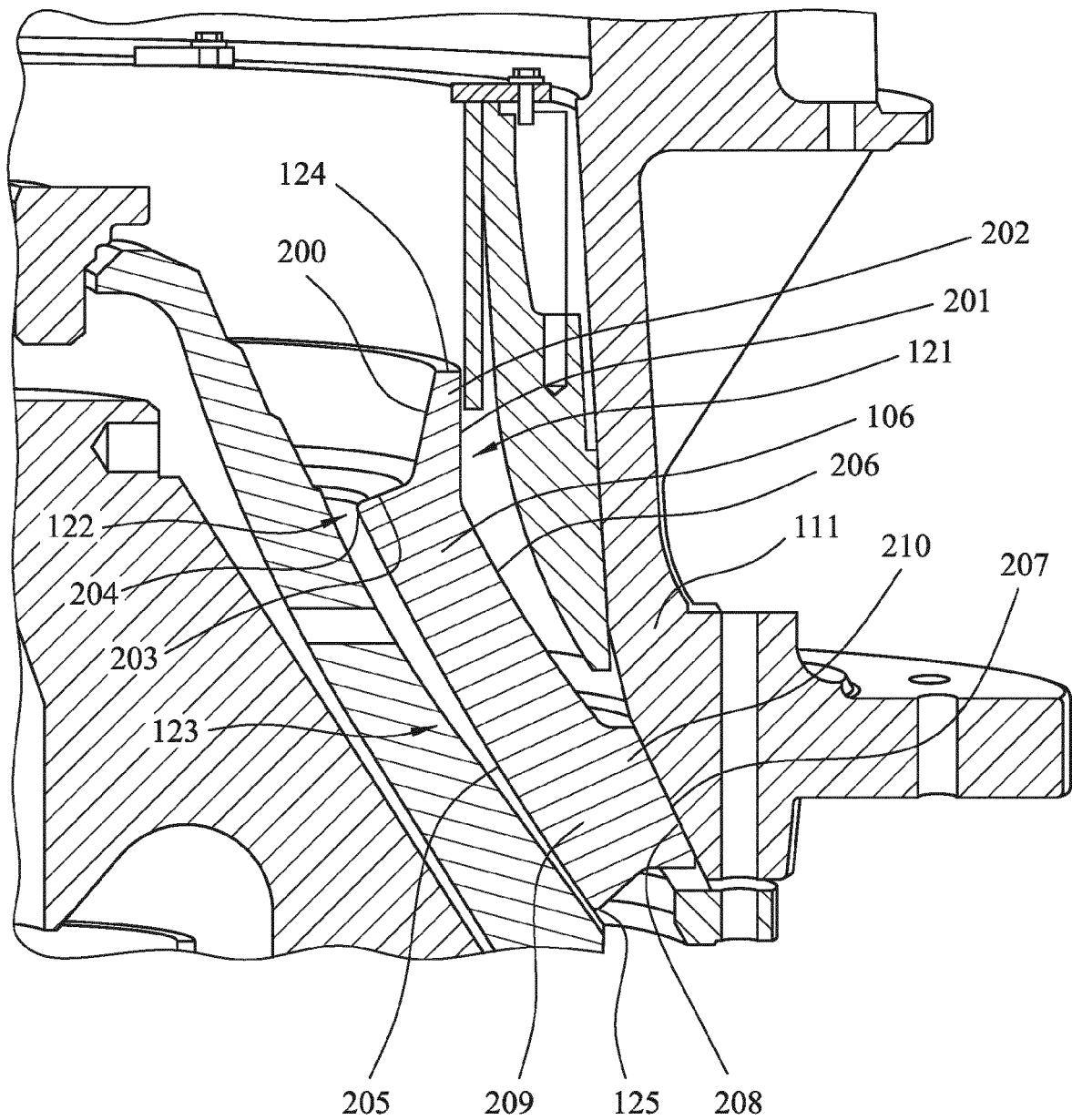


FIG. 2

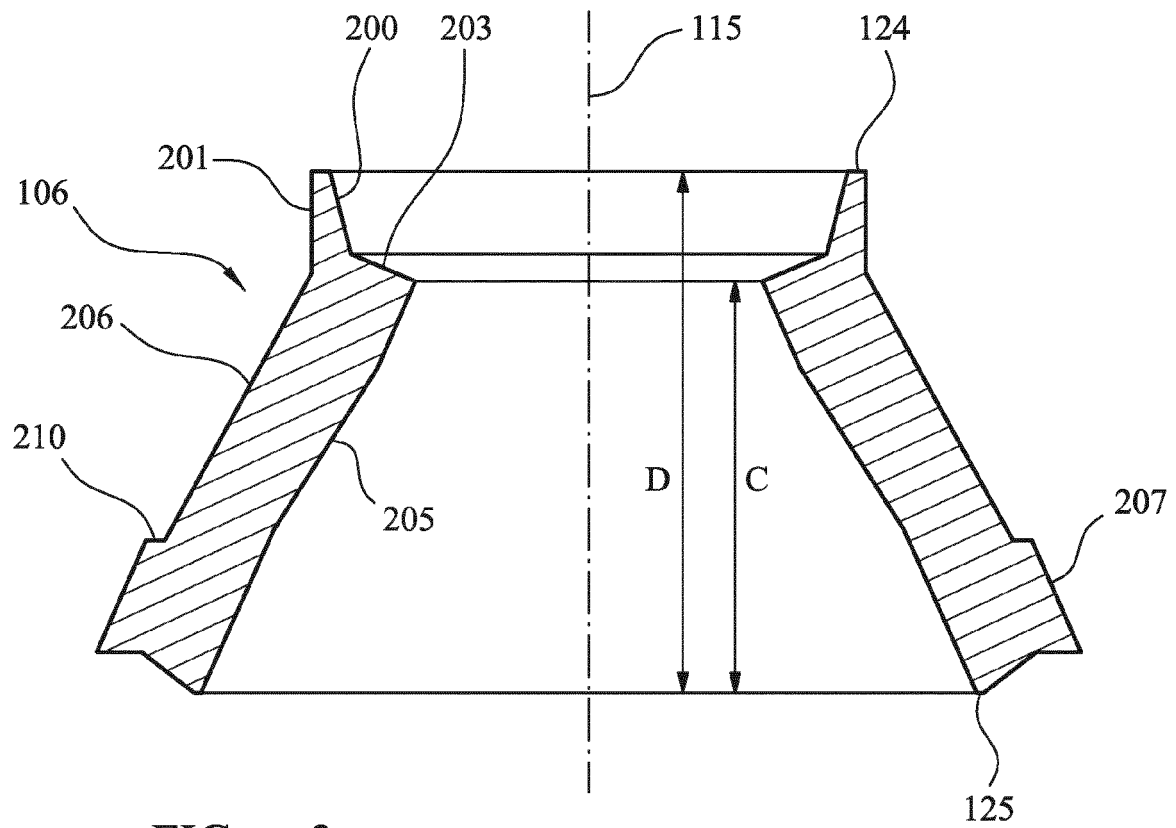


FIG. 3

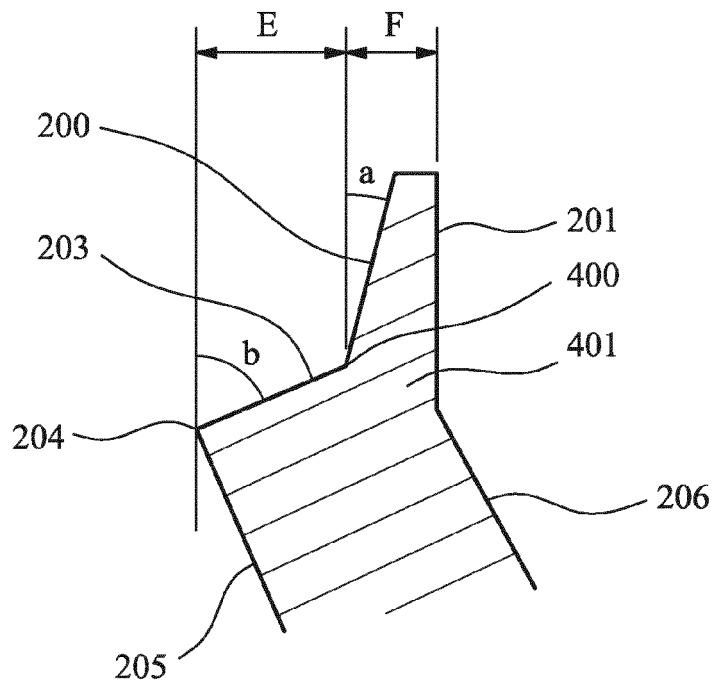


FIG. 4



EUROPEAN SEARCH REPORT

Application Number
EP 14 15 5672

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Place of search		Date of completion of the search	Examiner
Munich		9 July 2014	Flodström, Benny
CATEGORY OF CITED DOCUMENTS			
X : particularly relevant if taken alone		T : theory or principle underlying the invention	
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A : technological background		D : document cited in the application	
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**ANNEX TO THE EUROPEAN SEARCH REPORT
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