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(54) Title: GUIDE CAM ASSEMBLY FOR DIFFERENTIAL AND VARIABLE STROKE CYCLE ENGINES

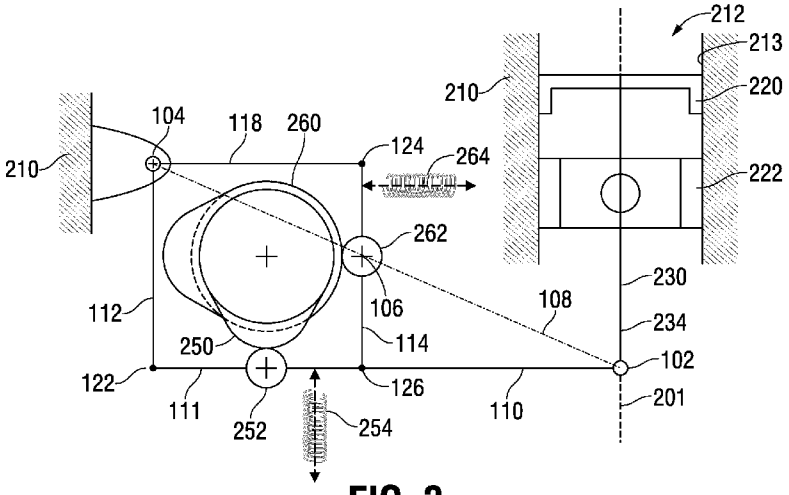

FIG. 3

(57) Abstract: An engine includes an engine shaft and a piston configured to reciprocate within a cylinder chamber having an axis, each piston having an first piston part and piston stem to move in unison with or separately from a second piston part to define piston strokes for different thermal functions of the engine. The engine further includes a linkage assembly having a first end coupled to the engine and a second end coupled to the piston stem defining a copy point, an actuator that engages the linkage assembly, and a guide cam that engages a guide cam follower on the linkage assembly. The actuator and the guide cam are operable to control motion of the linkage assembly to thereby define substantially linear movement of the copy point along the cylinder chamber axis.

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GUIDE CAM ASSEMBLY FOR DIFFERENTIAL AND VARIABLE STROKE CYCLE ENGINES

FIELD

Embodiments disclosed herein relate to internal combustion engines, and in particular, piston internal combustion engines. More particularly, embodiments disclosed herein relate to a guide cam assembly for guiding components of two-part pistons in differential and variable-stroke cycle internal combustion engines.

BACKGROUND AND SUMMARY

The internal combustion engine is an engine where the combustion of a fuel occurs with an oxidizer in a combustion chamber that is an integral part of the working fluid flow circuit. In an internal combustion engine the expansion of the high-temperature and high-pressure gases produced by combustion apply direct force to some component of the engine, typically a piston. This force moves the component over a distance, transforming chemical energy into useful mechanical energy.

In one aspect, embodiments disclosed herein relate to an engine having an engine shaft and a piston configured to reciprocate within a cylinder chamber having an axis, each piston having an first piston part and piston stem to move in unison with or separately from a second piston part to define piston strokes for different thermal functions of the engine. The engine further includes a linkage assembly having a first end coupled to the engine and a second end coupled to the piston stem defining a copy point, an actuator that engages the linkage assembly, and a guide cam that engages a guide cam follower on the linkage assembly. The actuator and the guide cam are operable to control motion of the linkage assembly to thereby define substantially linear movement of the copy point along the cylinder chamber axis.

BRIEF DESCRIPTION OF THE DRAWINGS

The invention is illustrated in the accompanying drawings wherein,

Figure 1 illustrates a schematic view of one embodiment of a guide cam assembly.

Figure 2 illustrates a top view of an embodiment of coaxial actuator and guide cams of a guide cam assembly.

Figure 3 illustrates a schematic view of an alternate embodiment of a guide cam assembly incorporating a pantographic linkage assembly.

Figure 4 illustrates a schematic view an alternate embodiment of a guide cam assembly incorporating a movable fulcrum.

Figure 5 illustrates a schematic view of an alternate embodiment of a guide cam assembly.

DETAILED DESCRIPTION

Embodiments disclosed herein relate to a guide cam assembly for guiding components of two-part pistons in differential and variable-stroke internal combustion engines. The engine typically includes an engine block having one or more cylinder bores and two-part pistons therein. Each two-part piston includes an upper or first piston part and a lower or second piston part which are separable from each other. The upper piston part is in sliding contact (or abutting) engagement with a respective cylinder bore wall and configured to at certain times engage the lower piston part. A piston stem is coupled at a first end to the upper piston part, and is hingedly (or pivotally) coupled at a second end to a linkage assembly. The hinged coupling (pivotal junction) may define a 'copy' point.

The guide cam assembly may include an actuator that engages the linkage assembly and thereby effects or controls vertical movement of the piston stem. In one embodiment, the actuator may be an actuator cam configured to engage an actuator cam follower on the linkage

assembly and thereby effect or control vertical movement of the piston stem. In turn, the piston stem effects or controls vertical movement of the upper piston part, which is also constrained by the cylinder bore wall. Alternatively, an electronic actuator may be used to effect or control vertical movement of the first piston part. Other actuation mechanisms may also be used including, but not limited to, an electromechanical actuator operable independently of the engine shaft, or a hydraulic actuator. Yet other actuation mechanisms may include means controlled electronically during engine operation such as electro-mechanical, electromagnetic, hydraulic, pneumatic or devices controlled via electronic circuit or solenoid.

The guide cam assembly further generally includes a guide cam configured to engage a guide cam follower at a different location on the linkage assembly and thereby control lateral movement of the piston stem. In turn, the piston stem controls lateral movement of the upper piston part, which is also constrained by the cylinder bore wall. One or more return mechanisms may be disposed at locations on the linkage assembly to bias the linkage assembly in a direction substantially opposite the mating engagement between respective cams and cam followers. A return mechanism may include a spring, a cam, an electro-mechanical actuator, a hydraulic actuator, a pneumatic actuator, or an electromagnetic actuator. In certain embodiments, multiple actuator and guide cams are coaxial, but are not required to be, and in other embodiments the multiple cams are not coaxial. Cam lobes or lobe profiles of any of the cams may be optimized to provide various different movements of the linkage assembly to in turn control movement of the copy point and piston stem, and thereby the first piston part.

Figure 1 illustrates a schematic view of one embodiment of a guide cam assembly. The variable-stroke cycle internal combustion engine typically includes an engine block 210 having one or more cylinder bores 212, and an upper or first piston part 220 located within each of the

one or more cylinder bores 212. The upper piston part 220 may be in sliding contact (or abutting) engagement with a respective cylinder bore wall 213. The upper piston part 220 is configured to at certain times engage a lower or second piston part 222. A piston stem 230 is coupled at a first end 232 to the upper piston part 220, and is hingedly (or pivotally) coupled at a second end 234 to a piston lever-link bar 110. The hinged coupling (pivotal junction) may define a 'copy' point 102.

The guide cam assembly includes a linkage assembly comprising a lever-link-bar 110 and fulcrum-link bar 112 coupled together at common ends 120. At an opposite end from this coupling 120, the lever-link bar 110 is coupled to the piston stem 230 at the copy point 102, and the fulcrum-link bar 112 is hingedly coupled to the engine block 210 at a first hinge junction 104. The hinged coupling (pivotal junction) defines an 'anchor' (or attachment) point 104. The guide cam assembly further includes 1) an actuator cam 250 configured to engage an actuator cam follower 252 on the lever-link bar 110 and thereby control vertical movement of the piston stem 230, which in turn controls vertical movement of the first piston part 220; and 2) a guide cam 260 configured to engage a guide cam follower 262 on the fulcrum-link bar 112 and thereby control lateral movement of the piston stem 230, which in turn controls lateral movement of the first piston part 220. One or more return mechanisms 254, 264 may be disposed at locations on the lever-link bar 110 and fulcrum-link bar 112, respectively, to bias each link in a direction substantially opposite the mating engagement between respective cams and cam followers. Figure 2 illustrates a top view of coaxial actuator and guide cams of the guide cam assembly arranged on a common shaft.

Figure 3 illustrates a schematic view of another embodiment of a guide cam assembly. The guide cam assembly incorporates a linkage assembly (*e.g.*, a four-bar-linkage) including a

portion 111 of the piston lever-link-bar 110, a fulcrum-link bar 112, a force-link bar 114, and a rocker-link bar 118. In defining and locating the four-bar-linkage, the linkage assembly may be hingedly coupled to the engine block 210 at a first hinge junction 104 of a first end of the fulcrum-link bar 112 and a first end of the rocker-link bar 118. The hinged coupling (pivotal junction) defines an 'anchor' (or attachment) point 104. The four-bar-linkage further includes a second hinge junction 122 of a second end of the fulcrum-link bar 112 and a first end of the portion 111 of the piston lever-link-bar 110, a third hinge junction 124 of a second end of the rocker-link bar 118 and a first end of the force-link bar 114, and a fourth hinge junction 126 of a second end of the force-link bar 114 and a second end of the portion 111 of the piston lever-link-bar 110.

The guide cam assembly further includes 1) an actuator cam 250 configured to engage an actuator cam follower 252 on the lever-link bar 110 and thereby control vertical movement of the piston stem 230, which in turn controls vertical movement of the first piston part 220; and 2) a guide cam 260 configured to engage a guide cam follower 262 on the force-link bar 114 and thereby control lateral movement of the piston stem 230, which in turn controls lateral movement of the first piston part 220. The guide cam follower 262 is coupled (for example rotatably or pivotally) to the force-link bar 114 at an "origin" point (or axis) 106. The "origin" point 106 is located at the intersection between the force-link bar 114 and an imaginary line – indicated by line 108 – defined between the 'copy' point 102 and the 'anchor' point 104. One or more return mechanisms 254, 264 may be disposed at locations on the lever-link bar 110 and force-link bar 114, respectively, to bias each link in a direction substantially opposite the mating engagement between respective cams and cam followers.

The four-bar-linkage of the guide apparatus 100 may be configured to form a pantographic assembly or apparatus. It will be understood by those skilled in the art that a pantographic assembly may be formed from mechanical linkages connected in a manner based on parallelograms, such that movement of one point of the assembly (for example, the "origin" point 106) produces respective (and possibly scaled) movements in a second point of the assembly (for example, the 'copy' point 102).

Figure 4 illustrates a schematic view of yet another embodiment of a guide cam assembly. The guide cam assembly incorporates a movable fulcrum at one end of the lever-link bar. A lever-link bar 110 is coupled at a first end to a cam follower 262 configured as the movable fulcrum, and at a second end to the first piston part 220 by way of the piston stem 230 at a copy point 102. The movable cam follower 262 may be configured to move in any direction. Preferably, the movable cam follower 262 may move in a direction substantially perpendicular to the cylinder axis 201. The guide cam assembly further includes 1) an actuator cam 250 configured to engage an actuator cam follower 252 on the lever-link bar 110 and thereby control vertical movement of the piston stem 230, which in turn controls vertical movement of the first piston part 220; and 2) a guide cam 260 configured to engage the movable cam follower 262 and thereby control lateral movement of the piston stem 230, which in turn controls lateral movement of the first piston part 220. A return mechanism 254 may be disposed at a location on the lever-link bar 110 to bias the lever-link bar 110 in a direction substantially opposite the mating engagement between the actuator cam 250 and cam follower 252.

Figure 5 illustrates a schematic view of yet another embodiment of a guide cam assembly. The guide cam assembly includes a linkage assembly comprising a lever-link-bar 110 and fulcrum-link bar 112 coupled together at common ends 120. At an opposite end from this

coupling 120, the lever-link bar 110 is coupled to the piston stem 230 at the copy point 102, and the fulcrum-link bar 112 is hingedly coupled to the engine block 210 at a first hinge junction 104. The hinged coupling (pivotal junction) defines an 'anchor' (or attachment) point 104. The guide cam assembly further includes an actuator 250 configured to engage the lever-link bar 110 and thereby control vertical movement of the piston stem 230, which in turn controls vertical movement of the first piston part 220. The actuator 250 may be any type of actuator, including but not limited to, an electronic actuator, an electromechanical actuator operable independently of the engine shaft, a hydraulic actuator, a pneumatic actuator, an electro-mechanical actuator, an electromagnetic actuator, an actuator controlled via electronic circuit or solenoid, or any other type capable of effecting movement of the linkage assembly. The guide cam assembly further includes a guide cam 260 configured to engage a guide cam follower 262 on the fulcrum-link bar 112 and thereby control lateral movement of the piston stem 230, which in turn controls lateral movement of the first piston part 220. The guide cam follower 262 may be rigidly coupled to the fulcrum-link bar 112 by a linkage 263. A return mechanism 254 may be disposed at a location on the lever-link bar 110 to bias the lever-link bar in a direction substantially opposite movement of the actuator 250. A return mechanism 264 may be disposed at a location on the fulcrum-link bar 112 to bias the fulcrum-link bar 112 in a direction substantially opposite the mating engagement between the guide cam 260 and the guide cam follower 262.

A method of operating a differential-stroke or variable-stroke reciprocating internal combustion engine, the engine having an engine shaft and a piston configured to reciprocate within a cylinder chamber having an axis, each piston having a first piston part and piston stem operable to move in unison with or separately from a second piston part to define piston strokes for different thermal functions of the engine, includes providing a linkage assembly having a first

end coupled to the engine and a second end coupled to the piston stem defining a copy point, an actuator that engages the linkage assembly, and a guide cam configured to engage a guide cam follower on the linkage assembly, wherein the actuator and guide cam are operable to control motion of the linkage assembly to thereby define substantially linear movement of the copy point along the cylinder chamber axis.

Reference throughout this specification to "one embodiment" or "an embodiment" or "certain embodiments" means that a particular feature, structure or characteristic described in connection with the embodiment is included in at least one embodiment of the present disclosure. Therefore, appearances of the phrases "in one embodiment" or "in an embodiment" or "in certain embodiments" in various places throughout this specification are not necessarily all referring to the same embodiment, but may. Furthermore, the particular features, structures or characteristics may be combined in any suitable manner, as would be apparent to one of ordinary skill in the art from this disclosure, in one or more embodiments.

In the claims below and the description herein, any one of the terms comprising, comprised of or which comprises is an open term that means including at least the elements/features that follow, but not excluding others. Therefore, the term comprising, when used in the claims, should not be interpreted as being limitative to the means or elements or steps listed thereafter. Any one of the terms including or which includes or that includes as used herein is also an open term that also means including at least the elements/features that follow the term, but not excluding others. Accordingly, including is synonymous with and means comprising.

It should be understood that the term "coupled," when used in the claims, should not be interpreted as being limitative to direct connections only. "Coupled" may mean that two or more

elements are either in direct physical, or that two or more elements are not in direct contact with each other but yet still cooperate or interact with each other. "Coupled" may mean a rigid coupling, hinged coupling, pivotal coupling, and others.

Although one or more embodiments of the present disclosure have been described in detail, it will be apparent to those skilled in the art that many embodiments taking a variety of specific forms and reflecting changes, substitutions and alterations may be made without departing from the scope of the invention as set out in the claims. The described embodiments illustrate the scope of the claims but do not restrict the scope of the claims.

CLAIMS

1. An engine having an engine shaft and a piston configured to reciprocate within a cylinder chamber having an axis, each piston having an first piston part and piston stem to move in unison with or separately from a second piston part to define piston strokes for different thermal functions of the engine, the engine comprising:

a linkage assembly having a first end coupled to the engine and a second end coupled to the piston stem defining a copy point;

an actuator that engages the linkage assembly; and

a guide cam that engages a guide cam follower coupled to the linkage assembly,

wherein the actuator and the guide cam are operable to control motion of the linkage assembly to thereby define substantially linear movement of the copy point along the cylinder chamber axis.

2. The engine of claim 1, wherein the actuator comprises an actuator cam that engages an actuator cam follower coupled to the linkage assembly.

3. The engine of claim 2, wherein the actuator cam and the guide cam are co-axial.

4. The engine of claim 2 or claim 3, further comprising a return mechanism configured to bias the linkage assembly in a direction substantially opposite the mating engagement between the actuator cam and the actuator cam follower.

5. The engine of any of claims 2 to 4, wherein the actuator cam is configured to effect vertical movement of the piston stem, which thereby effects vertical movement of the first piston part.

6. The engine of any preceding claim, wherein the guide cam follower is configured as a fulcrum point movable in a direction substantially perpendicular to the cylinder chamber axis.

7. The engine of claim 6, wherein the linkage assembly further comprises a four-bar linkage comprising a piston lever-link-bar, a fulcrum-link bar, a force-link bar, and a rocker-link-bar, wherein said four-bar-linkage is defined and located by:

a first hinge junction pivotally coupled to said engine and connecting a first end of said fulcrum-link bar and a first end of said rocker-link bar;

a second hinge junction connecting a second end of said fulcrum-link bar and a first end of said piston lever-link-bar;

a third hinge junction connecting a second end of said rocker-link bar and a first end of said force-link bar; and

a fourth hinge junction connecting a second end of said force-link bar and a location on said piston lever-link-bar.

8. The engine of claim 7, wherein the four-bar linkage defines a pantographic assembly.

9. The engine of any preceding claim, wherein the guide cam is configured to control lateral movement of the piston stem, which thereby controls lateral movement of the first piston part.

10. The engine of any of claims 1 to 3 and 5 to 9, further comprising a return mechanism configured to bias movement of the linkage assembly against one of the actuator or the guide cam.

11. The engine of claim 10, wherein the return mechanism comprises one of a spring, a cam, an electro-mechanical actuator, a hydraulic actuator, a pneumatic actuator, or an electromagnetic actuator.

12. The engine of claim 1, wherein the actuator comprises one of an electro-mechanical actuator, a hydraulic actuator, a pneumatic actuator, or an electromagnetic actuator.

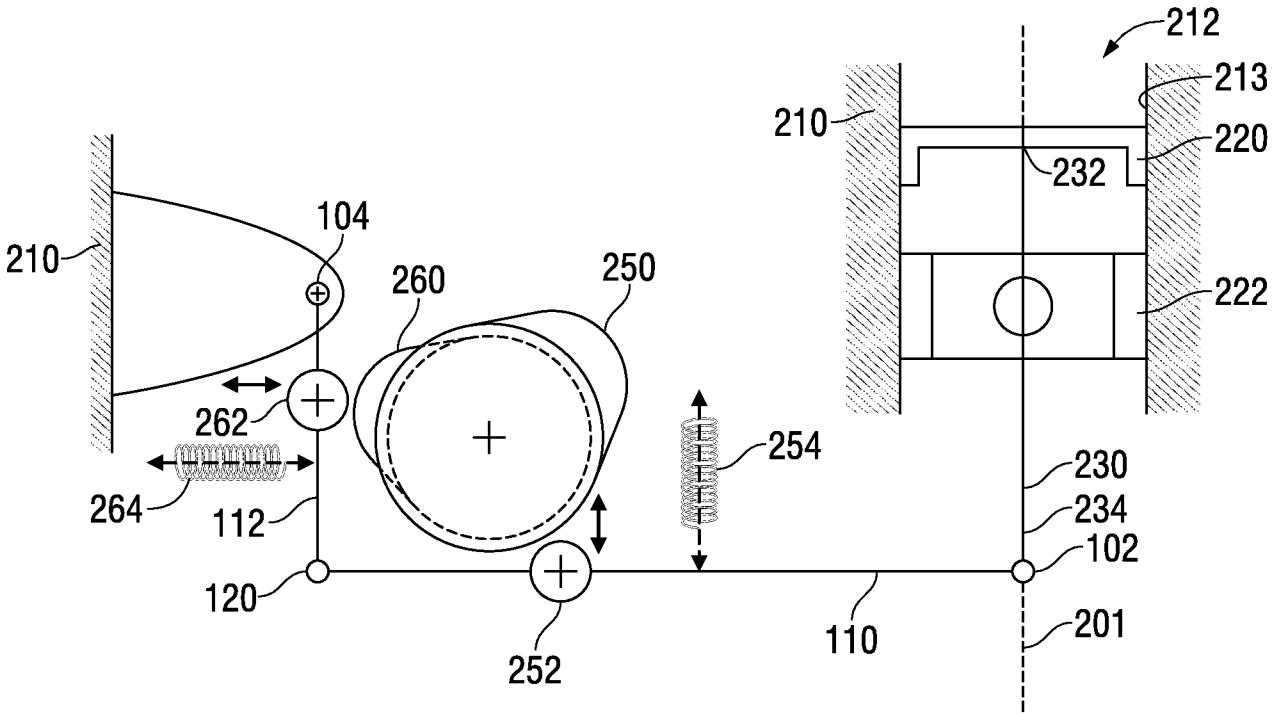
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FIG. 1

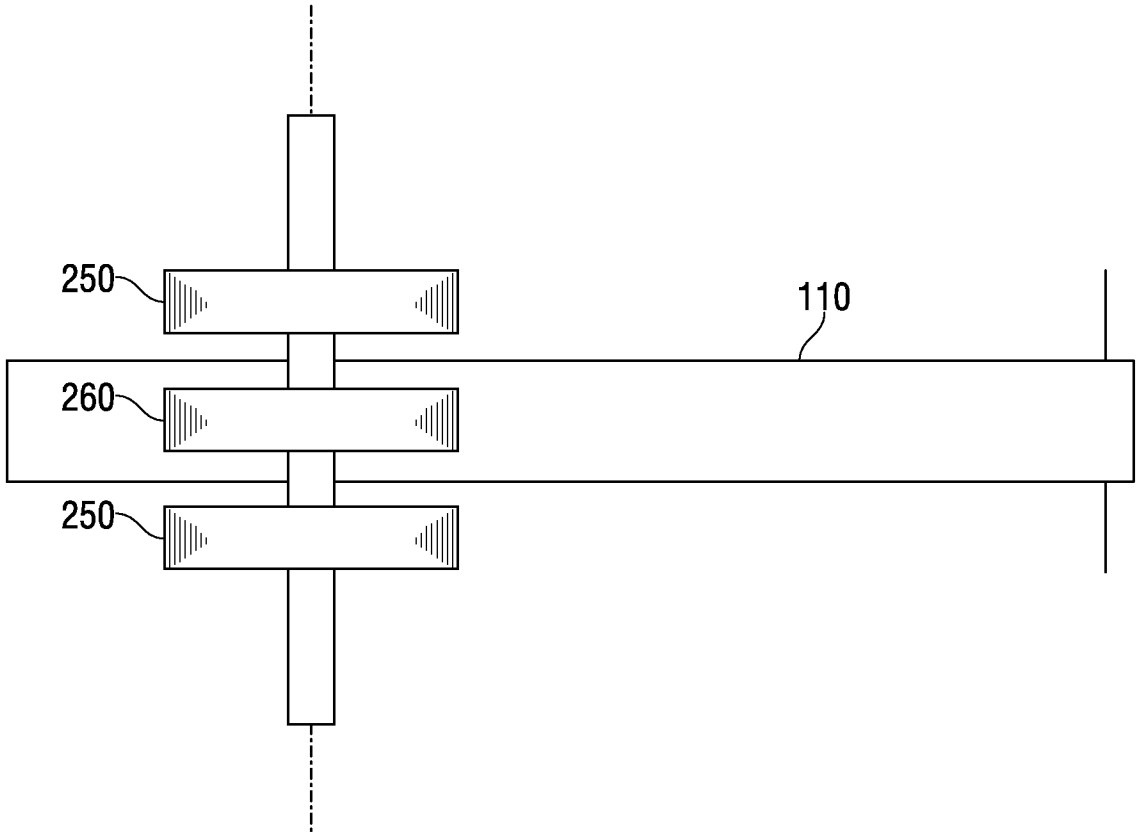
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FIG. 2

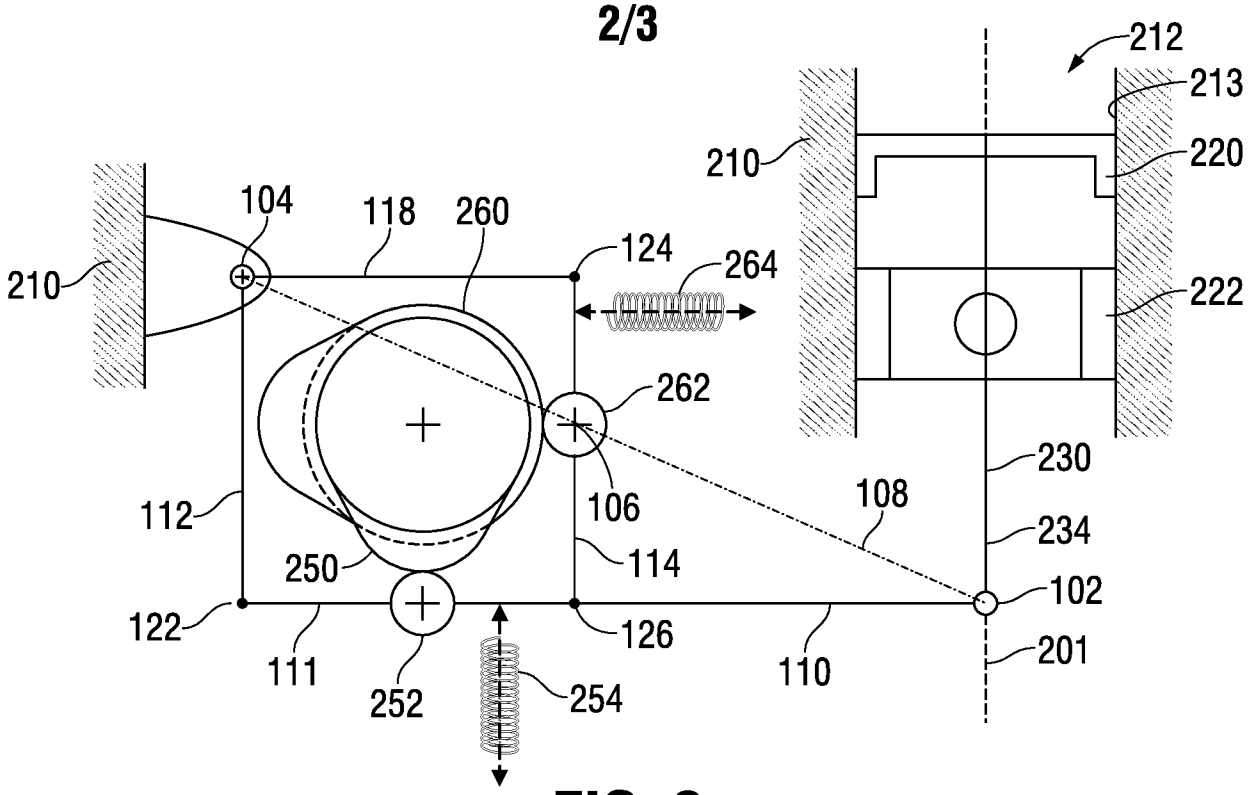
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FIG. 3

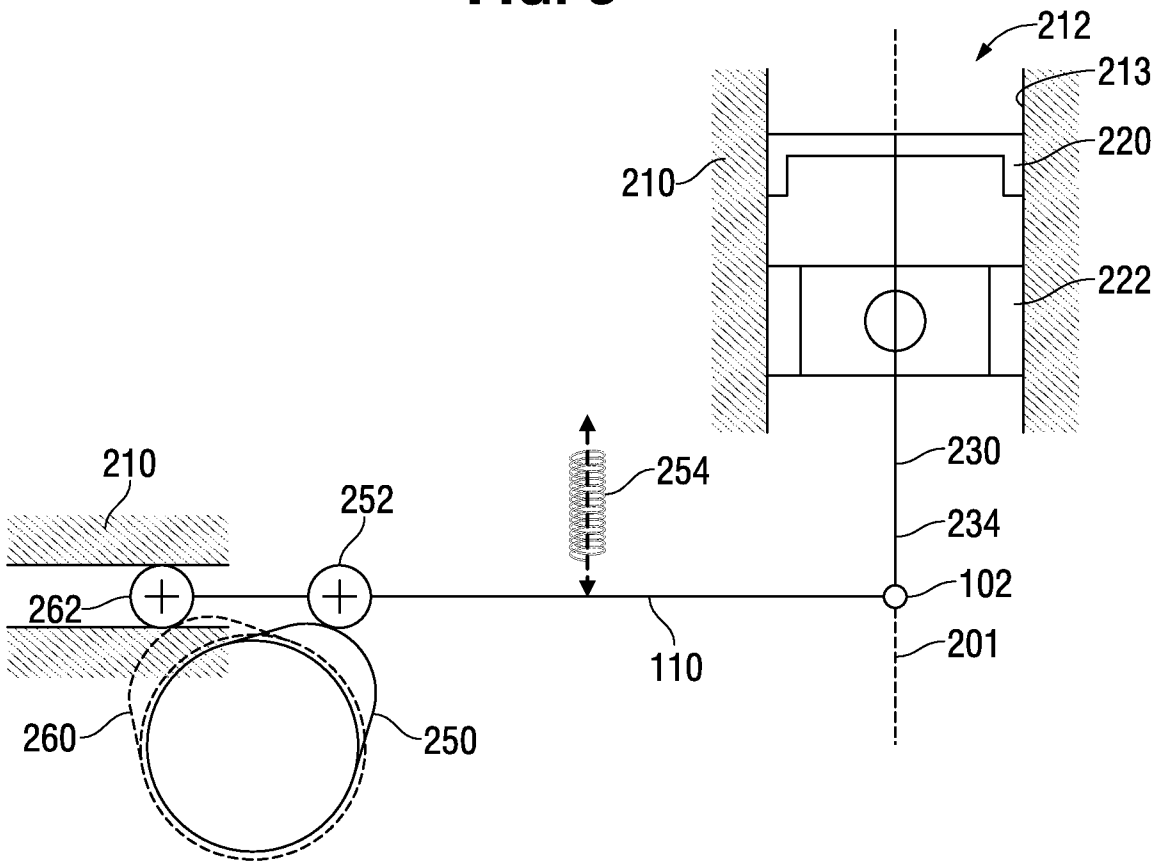
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FIG. 4

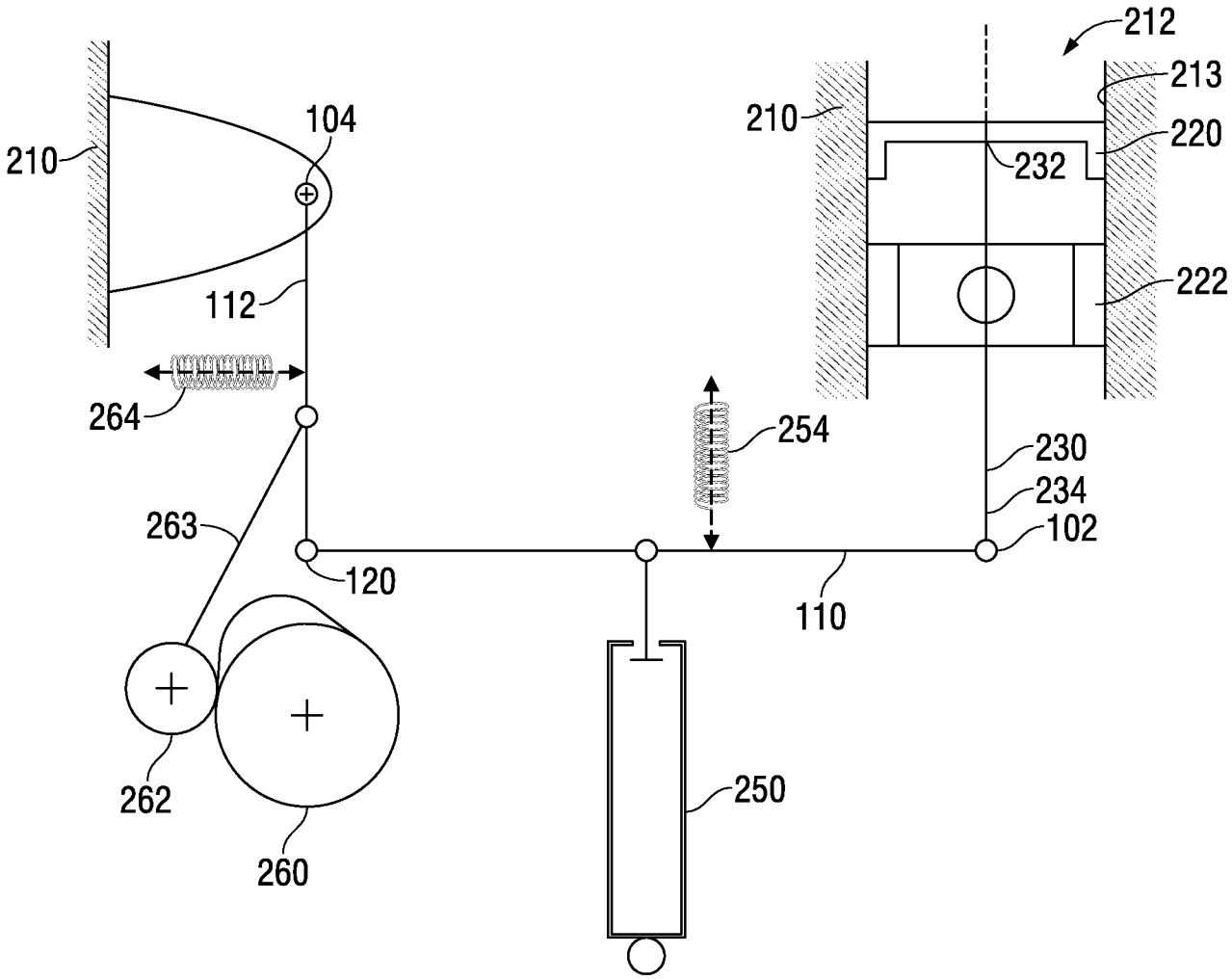
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FIG. 5

INTERNATIONAL SEARCH REPORT

International application No
PCT/GB2017/050895

A. CLASSIFICATION OF SUBJECT MATTER
INV. F02B75/32 F02B75/28 F02B75/02
ADD.

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)
F02B

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)
EPO-Internal, WPI Data

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
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Y	abstract; figures	2,4,6-8, 10-12

X	US 1 852 890 A (MACFARLANE ARTHUR D) 5 April 1932 (1932-04-05)	1
Y	abstract; figures	2,4,6-8, 10-12

A	US 2 222 367 A (KEOUGH JOHN J) 19 November 1940 (1940-11-19) the whole document	1-12

X	US 5 243 938 A (YAN MIIN J [US]) 14 September 1993 (1993-09-14) the whole document	1-4,7,8, 10,11

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[X] Further documents are listed in the continuation of Box C.

[X] See patent family annex.

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Date of the actual completion of the international search	Date of mailing of the international search report
23 June 2017	05/07/2017

Name and mailing address of the ISA/	Authorized officer
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INTERNATIONAL SEARCH REPORT

International application No
PCT/GB2017/050895

C(Continuation). DOCUMENTS CONSIDERED TO BE RELEVANT

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A	the whole document	2-12

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Information on patent family members

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