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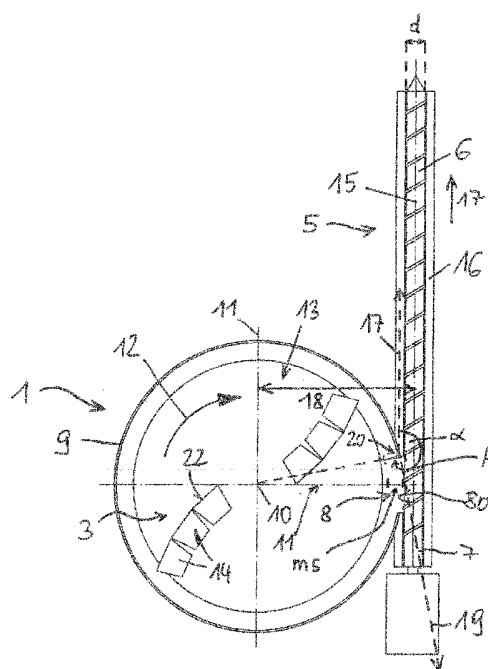
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[Fortsetzung auf der nächsten Seite]

(54) Title: APPARATUS FOR PROCESSING PLASTIC MATERIAL

(54) Bezeichnung : VORRICHTUNG ZUM AUFBEREITEN VON KUNSTSTOFFMATERIAL

Fig. 2



(57) Abstract: The invention relates to an apparatus for
preprocessing and subsequently conveying or plasticizing
plastics, comprising a container (1) with a mixing and/or
comminuting tool (3) that can rotate about a rotational axis
(10), wherein an opening (8) is formed in a lateral wall (9)
through which the plastic material can be discharged and a
conveyor (5) is provided with a screw (6) rotating in a
housing (16). The invention is characterized in that the
imaginary extension of the longitudinal axis (15) of the
conveyor (5) passes by the rotational axis (10) counter the
conveying direction (17), wherein the longitudinal axis (15)
is on the outlet side offset by a distance (18) in relation to
the radial (11) that is parallel to the longitudinal axis (15).
The smallest possible distance (m_s) between the tool (3) and
the screw (6) is defined by the following relationship: $m_s \leq k$
 $\cdot d + K$, wherein $d...$ is the diameter of the screw (6) in mm,
 K is a factor ranging from 20 to 100, in particular 20 to 80,
 k is a factor ranging from 0.03 to 0.4, in particular 0.04 to
0.25.

(57) Zusammenfassung: Die Erfindung betrifft
[Fortsetzung auf der nächsten Seite]

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**Veröffentlicht:**

— mit internationalem Recherchenbericht (Artikel 21 Absatz 3)

— vor Ablauf der für Änderungen der Ansprüche geltenden Frist; Veröffentlichung wird wiederholt, falls Änderungen eingehen (Regel 48 Absatz 2 Buchstabe h)

eine Vorrichtung zum Vorbehandeln und anschließenden Fördern oder Plastifizieren von Kunststoffen mit einem Behälter (1) mit einem um eine Drehachse (10) drehbaren Misch- und/oder Zerkleinerungswerkzeug (3), wobei in einer Seitenwand (9) eine Öffnung (8) ausgebildet ist, durch die das Kunststoffmaterial ausbringbar ist, wobei ein Förderer (5) vorgesehen ist, mit einer in einem Gehäuse (16) rotierenden Schnecke (6). Die Erfindung ist dadurch gekennzeichnet, dass die gedachte Verlängerung der Längsachse (15) des Förderers (5) entgegen der Förderrichtung (17) an der Drehachse (10) vorbeiführt, wobei die Längsachse (15) ablaufseitig zu der zur Längsachse (15) parallelen Radialen (11) um einen Abstand (18) versetzt ist, und dass der geringstmögliche Abstand (m_s) zwischen Werkzeug (3) und Schnecke (6) durch folgende Beziehung beschrieben wird: $m_s \leq k \cdot d + K$, wobei d ... Durchmesser der Schnecke (6) in mm, K ... Faktor im Bereich von 20 bis 100, insbesondere 20 bis 80, k ... Faktor im Bereich von 0,03 bis 0,4, insbesondere 0,04 bis 0,25, bedeutet.

Apparatus for the treatment of plastics material

The invention relates to an apparatus according to the preamble of Claim 1.

The prior art reveals numerous similar apparatuses of varying design, comprising a receiver or cutter compactor for the comminution, heating, softening and treatment of a plastics material to be recycled, and also, attached thereto, a conveyor or extruder for the melting of the material thus prepared. The aim here is to obtain a final product of the highest possible quality, mostly in the form of pellets.

By way of example, EP 123 771 or EP 303 929 describe apparatuses with a receiver and, attached thereto, an extruder, where the plastics material introduced into the receiver is comminuted through rotation of the comminution and mixing implements and is fluidized, and is simultaneously heated by the energy introduced. A mixture with sufficiently good thermal homogeneity is thus formed. This mixture is discharged after an appropriate residence time from the receiver into the screw-based extruder, and is conveyed and, during this process, plastified or melted. The arrangement here has the screw-based extruder approximately at the level of the comminution implements. The softened plastics particles are thus actively forced or stuffed into the extruder by the mixing implements.

Most of these designs, which have been known for a long time, however, are unsatisfactory in respect of the quality of the treated plastics material obtained at the outgoing end of the screw, and/or in respect of the quantitative output or throughput of the screw.

Critical to the end quality of the product are, firstly, the quality of the pretreated or softened polymer material that enters the conveyor or extruder from the cutter compactor, and, additionally, the situation at intake and on conveying or, where appropriate, extrusion. Relevant factors here include the length of the individual regions or zones of the screw, and also the screw parameters, such as, for example, screw thickness, flight depths, and so on.

In the case of the present cutter compactor/conveyor combinations, accordingly, there are particular circumstances, since the material which enters the conveyor is not introduced directly, without treatment and cold, but instead has already been pretreated in the cutter compactor, viz. heated, softened and/or partly crystallized, etc. This is a co-determining factor for the intake and for the quality of the material.

The two systems – that is, the cutter compactor and the conveyor – exert an influence on one another, and the outcomes of the intake and of the further conveying, and compaction, where appropriate, are heavily dependent on the pretreatment and the consistency of the material.

5 One important region, accordingly, is the interface between the cutter compactor and the conveyor, in other words the region in which the homogenized pretreated material is passed from the cutter compactor into the conveyor or extruder. On the one hand, this is a purely mechanical problem area, requiring the coupling to one another of two differently operating devices. Moreover, this interface is tricky for the polymer
10 material as well, since at this point the material is usually, close to the melting range, in a highly softened state, but is not allowed to melt. If the temperature is too low, then there are falls in the throughput and the quality; if the temperature is too high, and if unwanted melting occurs at certain places, then the intake becomes blocked.

 Furthermore, precise metering and feeding of the conveyor is difficult, since the
15 system is a closed system and there is no direct access to the intake; instead, the feeding of the material takes place from the cutter compactor, and therefore cannot be influenced directly, via a gravimetric metering device, for example.

 It is therefore critical to design this transition not only in a mechanically considered way, in other words with an understanding of the polymer properties, but at the same time
20 to consider the economics of the overall operation – in other words, high throughput and appropriate quality. The preconditions to be observed here are in some cases mutually contradictory.

 A feature shared by the apparatuses known from the prior art and mentioned in the introduction is that the direction of conveying or of rotation of the mixing and
25 comminution implements, and therefore the direction in which the particles of material circulate in the receiver, and the direction of conveying of the extruder, are in essence identical or have the same sense. This arrangement, selected intentionally, was the result of the desire to maximize stuffing of the material into the screw, or to force-feed the screw. This concept of stuffing the particles into the conveying screw or extruder screw in
30 the direction of conveying of the screw was also very obvious and was in line with the familiar thinking of the person skilled in the art, since it means that the particles do not have to reverse their direction of movement and there is therefore no need to exert any additional force for the change of direction. An objective here, and in further derivative developments, was always to maximize screw fill and to amplify this stuffing effect. By
35 way of example, attempts have also been made to extend the intake region of the

extruder in the manner of a cone or to curve the comminution implements in the shape of a sickle, so that these can act like a trowel in feeding the softened material into the screw. Displacement of the extruder, on the inflow side, from a radial position to a tangential position in relation to the container further amplified the stuffing effect, and increased the force with which the plastics material from the circulating implement was conveyed or forced into the extruder.

Apparatuses of this type are in principle capable of functioning, and they operate satisfactorily, although with recurring problems:

By way of example, an effect repeatedly observed with materials with low energy content, e.g. PET fibres or PET foils, or with materials which at a low temperature become sticky or soft, e.g. polylactic acid (PLA) is that when, intentionally, stuffing of the plastics material into the intake region of the extruder, under pressure, is achieved by components moving in the same sense, this leads to premature melting of the material immediately after, or else in, the intake region of the extruder. This firstly reduces the conveying effect of the extruder, and secondly there can also be some reverse flow of the said melt into the region of the cutter compactor or receiver, with the result that flakes that have not yet melted adhere to the melt, and in turn the melt thus cools and to some extent solidifies, with resultant formation of a clump or conglomerate made of to some extent solidified melt and of solid plastics particles. This causes blockage on the intake of the extruder and caking of the mixing and comminution implements. A further consequence is reduction of the throughput of the extruder, since adequate filling of the screw is no longer achieved. Another possibility here is that movement of the mixing and comminution implements is prevented. In such cases, the system normally has to be shut down and thoroughly cleaned.

Problems also occur with polymer materials which have already been heated in the cutter compactor up to the vicinity of their melting range. If overfilling of the intake region occurs here, the material melts and intake is impaired.

Problems are also encountered with fibrous materials that are mostly orientated and linear, with a certain amount of longitudinal elongation and low thickness or stiffness, for example plastics foils cut into strips. A main reason for this is that the elongate material is retained at the outflow end of the intake aperture of the screw, where one end of the strip protrudes into the receiver and the other end protrudes into the intake region. Since the mixing implements and the screw are moving in the same sense or exert the same conveying-direction component and pressure component on the material, both ends of the strip are subjected to tension and pressure in the same direction, and release

of the strip becomes impossible. This in turn leads to accumulation of the material in the said region, to a narrowing of the cross section of the intake aperture, and to poorer intake performance and, as a further consequence, to reduced throughput. The increased feed pressure in this region can moreover cause melting, and this in turn
5 causes the problems mentioned in the introduction.

Co-rotating cutter compactors of this kind have had a variety of extruders or conveyors attached to them, the results having in principle been entirely acceptable and attractive. The applicant, however, has performed comprehensive investigations for making still further improvements to the system as a whole.

10 It is therefore an object of the present invention to overcome the disadvantages mentioned and to improve an apparatus of the type described in the introduction in such a way as to permit problem-free intake of conventional materials by the screw, and also of those materials that are sensitive or strip-shaped, and to permit processing or treatment of these materials to give material of high quality, with high throughput, while making
15 efficient use of time, saving energy, and minimizing space requirement.

The characterizing features of Claim 1 achieve this object in an apparatus of the type mentioned in the introduction.

A first provision here is that the imaginary continuation of the central longitudinal axis of the conveyor, in particular extruder, if this has only a single screw, or the
20 longitudinal axis of the screw closest to the intake aperture, if the conveyor has more than one screw, in the direction opposite to the direction of conveying of the conveyor, passes, and does not intersect, the axis of rotation, where, on the outflow side, there is an offset distance between the longitudinal axis of the conveyor, if this has a single screw, or the longitudinal axis of the screw closest to the intake aperture, and the radius that is
25 associated with the container and that is parallel to the longitudinal axis and that proceeds outwards from the axis of rotation of the mixing and/or comminution implement in the direction of conveying of the conveyor.

The direction of conveying of the mixing implements and the direction of conveying of the conveyor are therefore no longer in the same sense, as is known from
30 the prior art, but instead are at least to a small extent in the opposite sense, and the stuffing effect mentioned in the introduction is thus reduced. The intentional reversal of the direction of rotation of the mixing and comminution implements in comparison with apparatuses known hitherto reduces the feed pressure on the intake region, and the risk

of overfilling decreases. In this way, excess material is not stuffed or trowelled with excess pressure into the intake region of the conveyor, but instead, in contrast, there is in fact in turn a tendency to remove excess material from that region, in such a way that although there is always sufficient material present in the intake region, the additional pressure exerted is small or almost zero. This method can provide adequate filling of the screw and constant intake of sufficient material by the screw, without any overfilling of the screw with, as a further consequence, local pressure peaks where the material could melt.

Melting of the material in the region of the intake is thus prevented, and operating efficiency is therefore increased, maintenance intervals are therefore lengthened, and downtime due to possible repairs and cleaning measures is reduced.

By virtue of the reduced feed pressure, displaceable elements which can be used in a known manner to regulate the degree of filling of the screw react markedly more sensitively, and the degree of filling of the screw can be adjusted with even greater precision. This makes it easier to find the ideal point at which to operate the system, in particular for relatively heavy materials, for example regrind made of high-density polyethylene (HDPE) or PET.

Surprisingly and advantageously it has moreover been found that operation in the opposite sense, according to the invention, improves intake of materials which have already been softened almost to the point of melting. In particular when the material is already in a doughy or softened condition, the screw cuts the material from the doughy ring adjacent to the container wall. In the case of a direction of rotation in the direction of conveying of the screw, this ring would instead be pushed onward, and removal of an outer layer by the screw would not be possible, with resultant impairment of intake. The reversal of the direction of rotation, according to the invention, avoids this.

Furthermore, the retention or accumulation phenomena formed in the case of the treatment of the materials which have been described above and are in strip form or fibrous can be resolved more easily, or do not occur at all, since, at the aperture edge situated in the direction of rotation of the mixing implements on the outflow side or downstream, the direction vector for the mixing implements and the direction vector for the conveyor point in almost opposite directions, or in directions that at least to a small extent have opposite sense, and an elongate strip cannot therefore become curved around, and retained by, the said edge, but instead becomes entrained again by the mixing vortex in the receiver.

The overall effect of the design according to the invention is that intake performance is improved and throughput is markedly increased. The stability and performance of the entire system made of cutter compactor and conveyor is thus increased.

5 The applicant has further found that by interposing a particular distance between the mixing and comminution implements and the screw it is possible to achieve advantageous effects which have a direct influence on the intake behaviour of the conveyor or extruder.

10 Accordingly, provision is also made, in accordance with the invention, for the smallest possible distance m_s between implement and screw to be described by the relationship $m_s \leq k * d + K$, where

d is the average diameter of the screw in mm in the region of the intake aperture,

K is a factor or a constant respectively in the range from 20 to 100, more particularly 20 to 80,

15 k is a factor in the range from 0.03 to 0.4, more particularly 0.04 to 0.25.

20 The distance m_s here is measured from the radially outermost point of the lowermost mixing and/or comminution implement, closest to the base, or blades and/or implements that protrude away and are provided there, up to a point situated on the envelope of the screw closest to the intake aperture. The distance m_s here runs along a radius which extends from the axis of the rotation of the container and through the aperture and the intake aperture up to the screw.

25 The distance m_s is the smallest possible distance which can be occupied by the tip of the implement relative to the screw when the tip of the implement rotates along past the screw. In the case of a tangential disposition of the screw to the container, in operation, the outermost implement tip moves past the aperture or intake aperture. In this case the distance changes continuously, and there is a minimum distance m_s .

30 This distance m_s , of the implement tip to the screw, is to be kept as small as possible, since this produces an improved intake behaviour and prevents a "pinching" of the material on intake into the screw. Nevertheless, sufficient tolerance distances must be ensured. If the distance becomes too great, the result is a deterioration in charging.

It has surprisingly been found that, because of the opposite direction of rotation of the mixing implements, the intake behaviour of the screw is not aggressive, and it is therefore possible, in the cutter compactor, to use implements that are more aggressive

and that introduce more energy into the material. Accordingly, the cutter compactor can be operated at relatively high temperature, a consequence of which is in turn better homogeneity with a reduced residence time. According to the invention, particularly good and effective introduction of energy is achieved by the particular space ratios in
5 combination with the reverse direction of rotation of the implements.

Another unexpected consequence of this type of combination of cutter compactor and extruder is improved melting performance of the material in an attached extruder since particles which are already greatly preheated pass into the screw. This provides compensation for possible inhomogeneity, and the material that passes from the
10 container into the screw housing and is then compressed and melted has high thermal and mechanical homogeneity. The final quality of the plastified or agglomerated material at the end of the extruder screw or of the agglomerating screw is correspondingly also very high, and it is possible to use screws which – because of the pretreatment and the intake – treat the polymer in a non-aggressive manner and introduce a particularly small
15 amount of shear into the material in order to melt the same.

Moreover, the constancy of throughput over time is higher, and/or the throughput performances are more uniform, and the intake operates reliably without problems associated with the filling of the screw.

Comparative experiment:

20 Comparative experiments were performed between a plant according to the prior art (Fig. 5) and a plant according to the invention (Fig. 6):

Both plants were plants comprising a cutter compactor having a diameter of 900 mm, connected tangentially to which was a 63 mm extruder (both plants were constructed in principle in the manner apparent from Figures 1 and 2, respectively). The
25 operating parameters were the same in each case. The material run was polyamide (PA) fibre. With the plant according to the invention, in contrast to the known plant, the direction of rotation of the implements in the cutter compactor was turned round counter-rotatingly, as provided in Figure 2. Moreover, the distance of the implements protruding above the carrier disc to the screw was set at 33 mm.

30 A measurement was made of the extruder discharge, in kg/h, over time. Likewise entered were the temperatures in the cutter compactor in the region of the intake aperture. The values were recorded every 30 minutes.

From this it can be seen that in the case of the prior-art plant, in comparison to the plant according to the invention, the throughput, first of all, exhibits a substantially greater

range of fluctuation, with values from 120 to 136 kg/h (Figure 5), in comparison to values of 136 to 147 kg/h (Figure 6). Secondly, the fluctuations are more frequent.

The plant according to the invention therefore has the more stable throughput over longer periods of time.

5 The temperature in the cutter compactor can be employed as a measure of the effects of the material introduced – for example, moisture content, fluctuation in bulk density, etc. In spite of similar fluctuations, the throughput remains substantially more constant than in the case of the prior art.

10 The difficulties associated with the filling of the screw are also apparent from the range of fluctuation of the temperature in the cutter compactor. If relatively long fibres, PA fibres for example, are retained around the intake in a downstream direction in the case of the known, co-rotating cutter compactor, there is a phasewise decrease in the intake and, as a further consequence, there is an increase in the throughput in the extruder and an increase in the temperature in the cutter compactor. A similar situation
15 occurs if there is overstuffing or overfeeding and the material in the extruder intake melts too early, with the material sticking in the intake.

Further advantageous embodiments of the invention are described via the following features:

20 According to one advantageous development, the smallest possible distance m_s is in the range from 15 mm to 150 mm.

The flight depth of the screw in the intake region of the conveyor or in the region of the intake aperture is in the range from 0.05 to 0.25 times, preferably 0.1 to 0.2 times, the diameter of the screw. This ensures a gentle but effective intake behaviour.

25 In this connection it is also advantageous if the clear width of the intake aperture and/or of the aperture is in the range from 0.8 to 3.5 times the diameter of the screw. The distance of the implements from the screw is also influenced by the length of the intake aperture.

The clear height of the intake aperture and/or of the aperture is preferably greater than or the same size as 0.2 times, preferably 0.3 times, the diameter of the screw.

30 The open area of the intake aperture and/or of the aperture is advantageously greater than $0.16 \cdot (d)^2$.

In one particularly advantageous embodiment, the ratio between the smallest

possible distance m_s and a distance A is in a range from 1 to 4.5. This distance A is measured between the radially outermost point of the lowermost mixing and/or comminution implement, closest to the base, or blades and/or implements that protrude down and are provided there, and a point lying on the envelope of the screw closest to the intake aperture. This point is situated in continuation of a radius which starts from the axis of rotation and passes through the aperture and the intake aperture, and at the same time contacts that point of the intake aperture that is situated furthest downstream, as viewed in the direction of conveying of the extruder. Viewed in the direction of conveying of the extruder, this point is situated on the downstream-sited edge of the intake aperture.

In this connection, particularly advantageous ratios have been found experimentally for different screw diameters:

Thus, for a screw diameter of 40 to 100 mm, the ratio of distance m_s to distance A is advantageously in a range from 1.02 to 1.75, or in a range from 1.18 to 2.6 for a diameter of 100 to 180 mm, and in a range from 1.4 to 4.5 for a diameter of 180 to 450 mm.

According to a further advantageous development of the invention, the conveyor is arranged on the receiver in such a way that the scalar product of the direction vector (direction vector that is associated with the direction of rotation) that is tangential to the circle described by the radially outermost point of the mixing and/or comminution implement or to the plastics material transported past the aperture and that is normal to a radius of the receiver, and that points in the direction of rotation or of movement of the mixing and/or comminution implement and of the direction vector that is associated with the direction of conveying of the conveyor at each individual point or in the entire region of the aperture or at each individual point or in the entire region immediately radially in front of the aperture is zero or negative. The region immediately radially in front of the aperture is defined as that region which is in front of the aperture and at which the material is just about to pass through the aperture but has not yet passed the aperture. The advantages mentioned in the introduction are thus achieved, and there is effective avoidance of all types of agglomeration in the region of the intake aperture, brought about by stuffing effects. In particular here, there is also no dependency on the spatial arrangement of the mixing implements and of the screw in relation to one another, and by way of example the orientation of the axis of rotation does not have to be normal to the basal surface or to the longitudinal axis of the conveyor or of the screw. The direction vector that is associated with the direction of rotation and the direction vector that is associated with the direction of conveying lie within a, preferably horizontal, plane, or in a

plane orientated so as to be normal to the axis of rotation.

In another advantageous formation, the angle included between the direction vector that is associated with the direction of rotation of the mixing and/or comminution implement and the direction vector that is associated with the direction of conveying of the conveyor is greater than or equal to 90° and smaller than or equal to 180° , where the angle is measured at the point of intersection of the two direction vectors at the edge that is associated with the aperture and that is situated upstream of the direction of rotation or of movement, in particular at the point that is on the said edge or on the aperture and is situated furthest upstream. This therefore describes the range of angles within which the conveyor must be arranged on the receiver in order to achieve the advantageous effects. In the entire region of the aperture or at each individual point of the aperture, the forces acting on the material are therefore orientated at least to a small extent in an opposite sense, or in the extreme case the orientation is perpendicular and pressure-neutral. At no point of the aperture is the scalar product of the direction vectors of the mixing implements and of the screw positive, and no excessive stuffing effect occurs even in a subregion of the aperture.

Another advantageous formation of the invention provides that the angle included between the direction vector that is associated with the direction of rotation or of movement and the direction vector that is associated with the direction of conveying is from 170° to 180° , measured at the point of intersection of the two direction vectors in the middle of the aperture. This type of arrangement is relevant by way of example when the conveyor is arranged tangentially on the cutter compactor.

In order to ensure that no excessive stuffing effect occurs, the distance, or the offset, between the longitudinal axis and the radius can advantageously be greater than or equal to half of the internal diameter of the housing of the conveyor or of the screw.

It can moreover be advantageous for these purposes to set the distance or offset between the longitudinal axis and the radius to be greater than or equal to 7%, or still more advantageously greater than or equal to 20%, of the radius of the receiver. In the case of conveyors with a prolonged intake region or with grooved bushing or with extended hopper, it can be advantageous for this distance or offset to be greater than or equal to the radius of the receiver. This is particularly true for cases where the conveyor is attached tangentially to the receiver or runs tangentially to the cross section of the container.

It is particularly advantageous if the longitudinal axis of the conveyor or of the screw or the longitudinal axis of the screw closest to the intake aperture runs tangentially

with respect to the inner side of the side wall of the container, or the inner wall of the housing does so, or the envelope of the screw does so, it being preferable that there is a drive connected to the end of the screw, and that the screw provides conveying, at its opposite end, to a discharge aperture, in particular an extruder head, arranged at the end
5 of the housing.

In the case of conveyors that are radially offset, but not arranged tangentially, it is advantageous to provide that the imaginary continuation of the longitudinal axis of the conveyor in a direction opposite to the direction of conveying, at least in sections, passes, in the form of a secant, through the space within the receiver.

10 It is advantageous to provide that there is immediate and direct connection between the aperture and the intake aperture, without substantial separation or a transfer section, e.g. a conveying screw. This permits effective and non-aggressive transfer of material.

The reversal of the direction of rotation of the mixing and comminution implements
15 circulating in the container can certainly not result from arbitrary action or negligence, and it is not possible – either in the known apparatuses or in the apparatus according to the invention – simply to allow the mixing implements to rotate in the opposite direction, in particular because the arrangement of the mixing and comminution implements is in a certain way asymmetrical or direction-oriented, and their action is therefore only single-
20 sided or unidirectional. If this type of equipment were to be rotated intentionally in the wrong direction, a good mixing vortex would not form, and there would be no adequate comminution or heating of the material. Each cutter compactor therefore has its unalterably prescribed direction of rotation of the mixing and comminution implements.

In this connection, it is particularly advantageous to provide that the manner of
25 formation, set-up, curvature and/or arrangement of the frontal regions or frontal edges that are associated with the mixing and/or comminution implements, act on the plastics material and point in the direction of rotation or of movement, differs when comparison is made with the regions that, in the direction of rotation or of movement, are at the rear or behind.

30 The implements and/or blades can either have been fastened directly on the shaft or preferably have been arranged on a rotatable implement carrier or, respectively, a carrier disc arranged in particular parallel to the basal surface, or have been formed therein or moulded onto the same, optionally as a single piece.

In principle, the effects mentioned are relevant not only to compressing extruders
35 or agglomerators but also to conveying screws that have no, or less, compressing effect.

Here again, local overfeed is avoided.

In another particularly advantageous formation, it is provided that the receiver is in essence cylindrical with a level basal surface and with, orientated vertically in relation thereto, a side wall which has the shape of the jacket of a cylinder. In another simple design, the axis of rotation coincides with the central axis of the receiver. In another advantageous formation, the axis of rotation or the central axis of the container have been orientated vertically and/or normally in relation to the basal surface. These particular geometries optimize intake performance, with an apparatus design that provides stability and simple construction.

In this connection it is also advantageous to provide that the mixing and/or comminution implement or, if a plurality of mutually superposed mixing and/or comminution implements have been provided, the lowest mixing and/or comminution implement closest to the base is arranged at a small distance from the basal surface, in particular in the region of the lowest quarter of the height of the receiver, and also that the aperture is similarly arranged. The distance here is defined and measured from the lowest edge of the aperture or of the intake aperture to the container base in the edge region of the container. There is mostly some rounding of the edge at the corner, and the distance is therefore measured from the lowest edge of the aperture along the imaginary continuations of the side wall downwards to the imaginary outward continuation of the container base. Distances with good suitability are from 10 to 400 mm.

The container does not necessarily have to have a cylindrical shape with circular cross section, even though this shape is advantageous for practical reasons and reasons of manufacturing technology. When container shapes that deviate from the cylindrical shape with circular cross section, examples being containers having the shape of a truncated cone or cylindrical containers which, in plan view, are elliptical or oval, a calculation is required for conversion to a cylindrical container which has circular cross section and the same volume capacity, on the assumption that the height of this imaginary container is the same as its diameter. Container heights here which are substantially higher than the resultant mixing vortex (after taking into account the distance required for safety) are ignored, since this excess container height is not utilized and it therefore has no further effect on the processing of the material.

The expression conveyor means mainly systems with screws that have non-compressing or decompressing effect, i.e. screws which have purely conveying effect, but also systems with screws that have compressing effect, i.e. extruder screws with agglomerating or plastifying effect.

The expressions extruder and extruder screw in the present text mean extruders or screws used for complete or partial melting of the material, and also extruders used to agglomerate, but not melt, the softened material. Screws with agglomerating effect subject the material to severe compression and shear only for a short time, but do not
 5 plastify the material. The outgoing end of the agglomerating screw therefore delivers material which has not been completely melted but which instead is composed of particles incipiently melted only at their surface, which have been caked together as if by sintering. However, in both cases the screw exerts pressure on the material and compacts it.

10 All of the examples described in the figure below depict conveyors with a single screw, for example single-screw extruders. However, it is also possible as an alternative to provide conveyors with more than one screw, for example twin- or multiscrew conveyors or twin- or multiscrew extruders, in particular with a plurality of identical screws, which at least have the same diameters d .

15 Further features and advantages of the invention are apparent from the description of the inventive examples below of the subject matter of the invention, which are not to be interpreted as restricting, and which the drawings depict diagrammatically and not to scale:

20 Figure 1 shows a vertical section through an apparatus according to the invention with extruder attached approximately tangentially.

Figure 2 shows a horizontal section through the embodiment of Figure 1.

Figure 3 shows another embodiment with minimal offset.

Figure 4 shows another embodiment with relatively large offset.

Figures 5 and 6 show test results.

25 Neither the containers, nor the screws nor the mixing implements are to scale, either themselves or in relation to one another, in the drawings. By way of example, therefore, the containers are in reality mostly larger, or the screws longer, than depicted here.

30 The advantageous cutter compactor-extruder combination depicted in **Figure 1** and **Figure 2** for the treatment or recycling of plastics material has a cylindrical container or cutter compactor or shredder 1 with circular cross section, with a level, horizontal basal surface 2 and with a vertical side wall 9 oriented normally thereto with the shape of a

cylinder jacket.

Arranged at a small distance from the basal surface 2, at most at about 10 to 20%, or optionally less, of the height of the side wall 9 – measured from the basal surface 2 to the uppermost edge of the side wall 9 – is an implement carrier 13 or a level
 5 carrier disc orientated parallel to the basal surface 2, which carrier or disc can be rotated, in the direction 12 of rotation or of movement indicated by an arrow 12, around a central axis 10 of rotation, which is simultaneously the central axis of the container 1. A motor 21, located below the container 1, drives the carrier disc 13. On the upper side of the carrier disc 13, blades or implements, e.g. cutter blades, 14 have been arranged, and
 10 together with the carrier disc 13 form the mixing and/or comminution implement 3.

As indicated in the diagram, the blades 14 are not arranged symmetrically on the carrier disc 13, but instead have a particular manner of formation, set-up or arrangement on their frontal edges 22 facing in the direction 12 of rotation or of movement, so that they can have a specific mechanical effect on the plastics material. The radially outermost
 15 edges of the mixing and comminution implements 3 reach a point which is relatively close to, about 5% of the radius 11 of the container 1 from, the inner surface of the side wall 9.

The container 1 has, near the top, a charging aperture through which the product to be processed, e.g. portions of plastics foils, is charged by way of example by means of a conveying device in the direction of the arrow. The container 1 can, as an alternative,
 20 be a closed container and capable of evacuation at least as far as an industrial vacuum, the material being introduced by way of a system of valves. The said product is received by the circulating mixing and/or comminution implements 3 and is raised to form a mixing vortex 30, where the product rises along the vertical side wall 9 and, approximately in the region of the effective container height H, falls back again inward and downward into the
 25 region of the centre of the container, under gravity. The effective height H of the container 1 is approximately the same as its internal diameter D. In the container 1, a mixing vortex is thus formed, in which the material is circulated in a vortex both from top to bottom and also in the direction 12 of rotation. By virtue of this particular arrangement of the mixing and comminution elements 3 or the blades 14, this type of apparatus can
 30 therefore be operated only with the prescribed direction 12 of rotation or movement, and the direction 12 of rotation cannot be reversed readily or without additional changes.

The circulating mixing and comminution implements 3 comminute and mix the plastics material introduced, and thereby heat and soften it by way of the mechanical frictional energy introduced, but do not melt it. After a certain residence time in the
 35 container 1, the homogenized, softened, doughy but not molten material is, as described

in detail below, removed from the container 1 through an aperture 8, passed into the intake region of an extruder 5, and received by a screw 6 there and subsequently melted.

At the level of the, in the present case single, comminution and mixing implement 3, the said aperture 8 is formed in the side wall 9 of the container 1, and the pretreated plastics material can be removed from the interior of the container 1 through this aperture. The material is passed to a single-screw extruder 5 arranged tangentially on the container 1, where the housing 16 of the extruder 5 has, situated in its jacket wall, an intake aperture 80 for the material to be received by the screw 6. This type of embodiment has the advantage that the screw 6 can be driven from the lower end in the drawing by a drive, depicted only diagrammatically, in such a way that the upper end of the screw 6 in the drawing can be kept free from the drive. The discharge aperture for the plastified or agglomerated plastics material conveyed by the screw 6 can therefore be arranged at the said right-hand end, e.g. in the form of an extruder head not depicted. The plastics material can therefore be conveyed without deflection by the screw 6 through the discharge aperture; this is not readily possible in the embodiments according to Figures 3 and 4.

There is connection for conveying of material or for transfer of material between the intake aperture 80 and the aperture 8, and in the present case this connection to the aperture 8 is direct and immediate and involves no prolonged intervening section and no separation. All that is provided is a very short transfer region.

In the housing 16, there is a screw 6 with compressing effect, mounted rotatably around its longitudinal axis 15. The longitudinal axis 15 of the screw 6 and that of the extruder 5 coincide. The extruder 5 conveys the material in the direction of the arrow 17. The extruder 5 is a conventional extruder known per se in which the softened plastics material is compressed and thus melted, and the melt is then discharged at the opposite end, at the extruder head.

The mixing and/or comminution implements 3 or the blades 14 are at approximately the same level as the central longitudinal axis 15 of the extruder 5. The outermost ends of the blades 14 have adequate separation from the flights of the screw 6.

In the embodiment according to Figures 1 and 2, the extruder 5 is, as mentioned, attached tangentially to the container 1, or runs tangentially in relation to its cross section. In the drawing, the imaginary continuation of the central longitudinal axis 15 of the extruder 5 or of the screw 6 in a direction opposite to the direction 17 of conveying of the extruder 5 towards the rear passes the axis 10 of rotation and does not intersect the

same. On the outflow side, there is an offset distance 18 between the longitudinal axis 15 of the extruder 5 or of the screw 6 and that radius 11 of the container 1 that is parallel to the longitudinal axis 15 and proceeds outwards from the axis 10 of rotation of the mixing and/or comminution implement 3 in the direction 17 of conveying of the conveyor 5. In
 5 the present case, the imaginary continuation of the longitudinal axis 15 of the extruder 5 towards the rear does not pass through the space within the container 1, but instead passes it at a short distance.

The distance 18 is somewhat greater than the radius of the container 1. There is therefore a slight outward offset of the extruder 5, or the intake region is somewhat
 10 deeper.

The expressions “opposite”, “counter-” and “in an opposite sense” here mean any orientation of the vectors with respect to one another which is not acute-angled, as explained in detail below.

In other words, the scalar product of a direction vector 19 which is associated with
 15 the direction 12 of rotation and the orientation of which is tangential to the circle described by the outermost point of the mixing and/or comminution implement 3 or tangential to the plastics material passing the aperture 8, and which points in the direction 12 of rotation or movement of the mixing and/or comminution implements 3, and of a direction vector 17 which is associated with the direction of conveying of the extruder 5 and which proceeds
 20 in the direction of conveying parallel to the central longitudinal axis 15 is everywhere zero or negative, at each individual point of the aperture 8 or in the region radially immediately in front of the aperture 8, and is nowhere positive.

In the case of the intake aperture in Figures 1 and 2, the scalar product of the direction vector 19 for the direction 12 of rotation and of the direction vector 17 for the
 25 direction of conveying is negative at every point of the aperture 8.

The angle α between the direction vector 17 for the direction of conveying and the direction vector for the direction 19 of rotation, measured at the point 20 that is associated with the aperture 8 and situated furthest upstream of the direction 12 of rotation, or at the edge associated with the aperture 8 and situated furthest upstream, is approximately
 30 maximally about 170° .

As one continues to proceed downwards along the aperture 8 in Figure 2, i.e. in the direction 12 of rotation, the oblique angle between the two direction vectors continues to increase. In the centre of the aperture 8, the angle between the direction vectors is about 180° and the scalar product is maximally negative, and further downwards from
 35 there the angle indeed becomes $> 180^\circ$ and the scalar product in turn decreases, but still

remains negative. However, these angles are no longer termed angles α , since they are not measured at point 20.

An angle β , not included in the drawing in Figure 2, measured in the centre of the aperture 8, between the direction vector for the direction 19 of rotation and the direction
5 vector for the direction 17 of conveying is about 178° to 180° .

The apparatus according to Figure 2 represents the first limiting case or extreme value. This type of arrangement can provide a very non-aggressive stuffing effect or a particularly advantageous feed, and this type of apparatus is particularly advantageous for sensitive materials which are treated in the vicinity of the melting range, or for product
10 in the form of long strips.

The smallest possible distance m_s between implement and screw, measured from the radially outermost point or from the tip of the blade 14 that is situated the furthest outwards – the implements or blades 14 protrude above the carrier disc 13 or in front of it – or from the circle defined thereby, up to the envelope of the screw 6, is drawn in by
15 way of example in Figure 2. The distance m_s is situated in essence centrally to the length of the inlet aperture 80 and on the radius 11 which is directed at an angle of 90° to the longitudinal axis 15 of the extruder 5. If the implement 3 moves further downwards or upwards, then the distance becomes larger again. At the edges of the intake aperture 80, the distance is at a maximum when the intake aperture 80 is situated with its length, as in
20 Figure 2, symmetrical to the 90° radius 11.

If the implement 3 or the outermost tip of the blade 14 is moved in the direction of the edge 20' of the intake aperture 80, then the distance becomes greater, and at the exact point where the radials 11 contact the edge 20' or the point 20, the distance A between implement tip and screw 6 is measured.

25 With particular screw diameters, it is advantageous to select the ratios for the distances m_s to A accordingly.

In Figures 3 and 4, the distances m_s and A are not drawn in. Here, the distances m_s and A are also dependent on the design of the end face of the screw 6. Figures 3 and 4 serve primarily for illustrating the connection possibilities of the extruder.

30 **Figure 3** shows an alternative embodiment in which the extruder 5 is attached to the container 1 not tangentially but instead by its end 7. The screw 6 and the housing 16 of the extruder 5 have been adapted in the region of the aperture 8 to the shape of the inner wall of the container 1, and have been offset backwards so as to be flush. No part of the extruder 5 protrudes through the aperture 8 into the space within the container 1.

The distance 18 here corresponds to about 5 to 10% of the radius 11 of the container 1 and to about half of the internal diameter d of the housing 16. This embodiment therefore represents the second limiting case or extreme value with the smallest possible offset or distance 18, where the direction 12 of rotation or of movement of the mixing and/or comminution implements 3 is at least slightly opposite to the direction 17 of conveying of the extruder 5, and specifically across the entire area of the aperture 8.

The scalar product in Figure 3 at that threshold point 20 situated furthest upstream is precisely zero, where this is the point located at the edge that is associated with the aperture 8 and situated furthest upstream. The angle α between the direction vector 17 for the direction of conveying and the direction vector for the direction 19 of rotation, measured at point 20 in Figure 3, is precisely 90° . If one proceeds further downwards along the aperture 8, i.e. in the direction 12 of rotation, the angle between the direction vectors becomes ever greater and becomes an oblique angle $> 90^\circ$, and at the same time the scalar product becomes negative. However, at no point, or in no region of the aperture 8, is the scalar product positive, or the angle smaller than 90° . No local overfeed can therefore occur even in a subregion of the aperture 8, and no detrimental excessive stuffing effect can occur in a region of the aperture 8.

This also represents a decisive difference in relation to a purely radial arrangement, since there would be an angle $\alpha < 90^\circ$ at point 20 or at the edge 20' in a fully radial arrangement of the extruder 5, and those regions of the aperture 8 situated, in the drawing, above the radius 11 or upstream thereof or on the inflow side thereof would have a positive scalar product. It would thus be possible for locally melted plastics product to accumulate in these regions.

Figure 4 depicts another alternative embodiment in which the extruder 5 is somewhat further offset than in Figure 3 on the outflow side, but still not tangentially as in Figures 1 and 2. In the present case, as also in Figure 3, the rearward imaginary continuation of the longitudinal axis 15 of the extruder 5 passes through the space within the container 1 in the manner of a secant. As a consequence of this, the aperture 8 is – measured in the circumferential direction of the container 1 – wider than in the embodiment according to Figure 3. The distance 18 is also correspondingly greater than in Figure 3, but somewhat smaller than the radius 11. The angle α measured at point 20 is about 150° , and the stuffing effect is therefore reduced in comparison with the apparatus of Figure 3; this is more advantageous for certain sensitive polymers. The inner wall of the housing 16 or the right-hand-side inner edge, as seen from the

container 1, is tangential to the container 1, and therefore, unlike in Figure 3, there is no oblique transitional edge. At this point that is associated with the aperture 8 and situated furthest downstream, on the extreme left-hand side in Figure 4, the angle is about 180° .

Patent claims:

1. Apparatus for the pretreatment and subsequent conveying, plastification or agglomeration of plastics, in particular of thermoplastics waste for recycling purposes, with a container (1) for the material to be processed, where the arrangement has, in the container (1), at least one mixing and/or comminution implement (3) which rotates around an axis (10) of rotation and which is intended for the mixing, heating and optionally comminution of the plastics material, where an aperture (8) through which the pretreated plastics material can be removed from the interior of the container (1) is formed in a side wall (9) of the container (1) in the region of the level of the, or of the lowest, mixing and/or comminution implement (3) that is closest to the base, where at least one conveyor (5), in particular one extruder (5), is provided to receive the pretreated material, and has at least one screw (6) which rotates in a housing (16) and which in particular has plastifying or agglomerating action, where the housing (16) has, located at its end (7) or in its jacket wall, an intake aperture (80) for the material to be received by the screw (6), and there is connection between the intake aperture (80) and the aperture (8),

characterized in that

the imaginary continuation of the central longitudinal axis (15) of the conveyor (5) or of the screw (6) closest to the intake aperture (80), in a direction opposite to the direction (17) of conveying of the conveyor (5), passes, and does not intersect, the axis (10) of rotation, there being, on the outflow side or in the direction (12) of rotation or of movement of the mixing and/or comminution implement (3), an offset distance (18) between the longitudinal axis (15) of the conveyor (5) or of the screw (6) closest to the intake aperture (80), and the radius (11) of the container (1) that is parallel to the longitudinal axis (15) and that proceeds outwards from the axis (10) of rotation of the mixing and/or comminution implement (3) in the direction (17) of conveying of the conveyor (5), and in that the smallest possible distance (ms) between implement (3) and screw (6) is described by the following relationship:

$$ms \leq k * d + K$$

where

d is the diameter of the screw (6) in mm,

K is a factor in the range from 20 to 100, more particularly 20 to 80,

k is a factor in the range from 0.03 to 0.4, more particularly 0.04 to 0.25,

the distance (ms) being measured between the radially outermost point of the lowest mixing and/or comminution implement (3) closest to the base, or implements and/or blades (14) provided there, and a point lying on the envelope of the screw (6) lying closest to the intake aperture (80), more specifically along a radius (11) which extends from the axis (10) of rotation and passes through the aperture (8) and the intake aperture (80).

2. Apparatus according to Claim 1, characterized in that the distance (ms) is in the range from 15 mm to 50 mm.
3. Apparatus according to Claim 1 or 2, characterized in that the flight depth of the screw (6) in the intake region of the conveyor (5) or in the region of the intake aperture (80) is in the range from 0.05 to 0.25, preferably from 0.1 to 0.2, times the diameter (d) of the screw (6).
4. Apparatus according to any of Claims 1 to 3, characterized in that the clear width of the intake aperture (80) and/or of the aperture (8) is in the range from 0.8 to 3.5 times the diameter (d) of the screw (6).
5. Apparatus according to any of Claims 1 to 4, characterized in that the clear height of the intake aperture (80) and/or of the aperture (8) is greater than or equal to 0.2 times, preferably 0.3 times, the diameter (d) of the screw (6).
6. Apparatus according to any of Claims 1 to 5, characterized in that the open area of the intake aperture (80) and/or of the aperture (8) is greater than $0.16 \cdot (d)^2$.
7. Apparatus according to any of Claims 1 to 6, characterized in that, especially when the screw (6) is disposed tangentially to the container (1), the ratio between the smallest possible distance (ms) and a distance (A) is in a range from 1 to 4.5, where the distance (A) is measured between the radially outermost point of the lowest mixing and/or comminution implement (3) closest to the base, or implements and/or blades (14) provided there, and a point lying on the envelope of the screw (6) lying closest to the intake aperture (80), more specifically along a radius (11) which extends from the axis (10) of rotation and passes through the aperture (8) and the intake aperture (80) and at the same time contacts that point (20) of the intake aperture (80) that is situated furthest downstream as viewed

in the direction (17) of conveying of the extruder (5).

8. Apparatus according to Claim 7, characterized in that, for a screw (6) diameter (d) of from 40 to 100 mm, the ratio of distance (ms) to distance (A) is in a range from 1.02 to 1.75.
9. Apparatus according to Claim 7, characterized in that, for a screw (6) diameter (d) of from 100 to 180 mm, the ratio of distance (ms) to distance (A) is in a range from 1.18 to 2.6.
10. Apparatus according to Claim 7, characterized in that, for a screw (6) diameter (d) of from 180 to 450 mm, the ratio of distance (ms) to distance (A) is in a range from 1.4 to 4.5.
11. Apparatus according to any of Claims 1 to 10, characterized in that, for a conveyor (5) in contact with the container (1), the scalar product of the direction vector that is associated with the direction (19) of rotation and that is tangential to the circle described by the radially outermost point of the mixing and/or comminution implement (3) or that is tangential to the plastics material transported past the aperture (8) and that is normal to a radius (11) of the container (1), and that points in the direction (12) of rotation or of movement of the mixing and/or comminution implement (3) and of the direction vector (17) that is associated with the direction of conveying of the conveyor (5) at each individual point or in the entire region of the aperture (8) or immediately radially in front of the aperture (8) is zero or negative.
12. Apparatus according to any of Claims 1 to 10, characterized in that the angle (α) included between the direction vector that is associated with the direction (19) of rotation of the radially outermost point of the mixing and/or comminution implement (3) and the direction vector (17) that is associated with the direction of conveying of the conveyor (5) is greater than or equal to 90° and smaller than or equal to 180° , measured at the point of intersection of the two direction vectors (17, 19) at the inflow-side edge that is associated with the aperture (8) and that is situated upstream in relation to the direction (12) of rotation or of movement of the mixing and/or comminution implement (3), in particular at the point (20) that is on the said edge or on the aperture (8) and is situated furthest upstream.
13. Apparatus according to any of Claims 1 to 12, characterized in that the angle (β) included between the direction vector (19) that is associated with the direction (12)

of rotation or of movement and the direction vector (17) that is associated with the direction of conveying of the conveyor (5) is from 170° to 180° , measured at the point of intersection of the two direction vectors (17, 19) in the middle of the aperture (8).

14. Apparatus according to any of Claims 1 to 13, characterized in that the distance (18) is greater than or equal to half of the internal diameter of the housing (16) of the conveyor (5) or of the screw (6), and/or greater than or equal to 7%, preferably greater than or equal to 20%, of the radius of the container (1), or in that the distance (18) is greater than or equal to the radius of the container (1).
15. Apparatus according to any of Claims 1 to 14, characterized in that the imaginary continuation of the longitudinal axis (15) of the conveyor (5) in a direction opposite to the direction of conveying is arranged in the manner of a secant in relation to the cross section of the container (1), and, at least in sections, passes through the space within the container (1).
16. Apparatus according to any of Claims 1 to 15, characterized in that the conveyor (5) is attached tangentially to the container (1) or runs tangentially in relation to the cross section of the container (1), or in that the longitudinal axis (15) of the conveyor (5) or of the screw (6) or the longitudinal axis of the screw (6) closest to the intake aperture (80) runs tangentially with respect to the inner side of the side wall (9) of the container (1), or the inner wall of the housing (16) does so, or the envelope of the screw (6) does so, where it is preferable that there is a drive connected to the end (7) of the screw (6), and that the screw provides conveying, at its opposite end, to a discharge aperture, in particular an extruder head, arranged at the end of the housing (16).
17. Apparatus according to any of Claims 1 to 16, characterized in that there is immediate and direct connection between the aperture (8) and the intake aperture (80), without substantial separation, in particular without transfer section or conveying screw.
18. Apparatus according to any of Claims 1 to 17, characterized in that the mixing and/or comminution implement (3) comprises implements and/or blades (14) which, in the direction (12) of rotation or of movement, have a comminuting, cutting and heating effect on the plastics material, where the implements and/or blades (14) are preferably arranged or formed on or at a rotatable implement carrier (13) which is in

particular a carrier disc (13) and which is in particular arranged parallel to the basal surface (2).

19. Apparatus according to any of Claims 1 to 18, characterized in that the manner of formation, set-up, curvature and/or arrangement of the frontal regions or frontal edges (22) that are associated with the mixing and/or comminution implements (3) or with the blades (14), act on the plastics material and point in the direction (12) of rotation or of movement, differs when comparison is made with the regions that, in the direction (12) of rotation or of movement, are at the rear or behind.
20. Apparatus according to any of Claims 1 to 19, characterized in that the container (1) is in essence cylindrical with circular cross section and with a level basal surface (2) and with, orientated vertically in relation thereto, a side wall (9) which has the shape of the jacket of a cylinder, and/or the axis (10) of rotation of the mixing and/or comminution implements (3) coincides with the central axis of the container (1), and/or the axis (10) of rotation or the central axis are orientated vertically and/or normally in relation to the basal surface (2).
21. Apparatus according to any of Claims 1 to 20, characterized in that the lowest implement carrier (13) or the lowest of the mixing and/or comminution implements (3) and/or the aperture (8) are arranged close to the base at a small distance from the basal surface (2), in particular in the region of the lowest quarter of the height of the container (1), preferably at a distance of from 10 mm to 400 mm from the basal surface (2).
22. Apparatus according to any of Claims 1 to 21, characterized in that the conveyor (5) is a single-screw extruder (6) with a single compression screw (6), or is a twin- or multiscrew extruder, where the diameters d of the individual screws (6) are all identical.

Vienna, October 14, 2011

Fig. 1

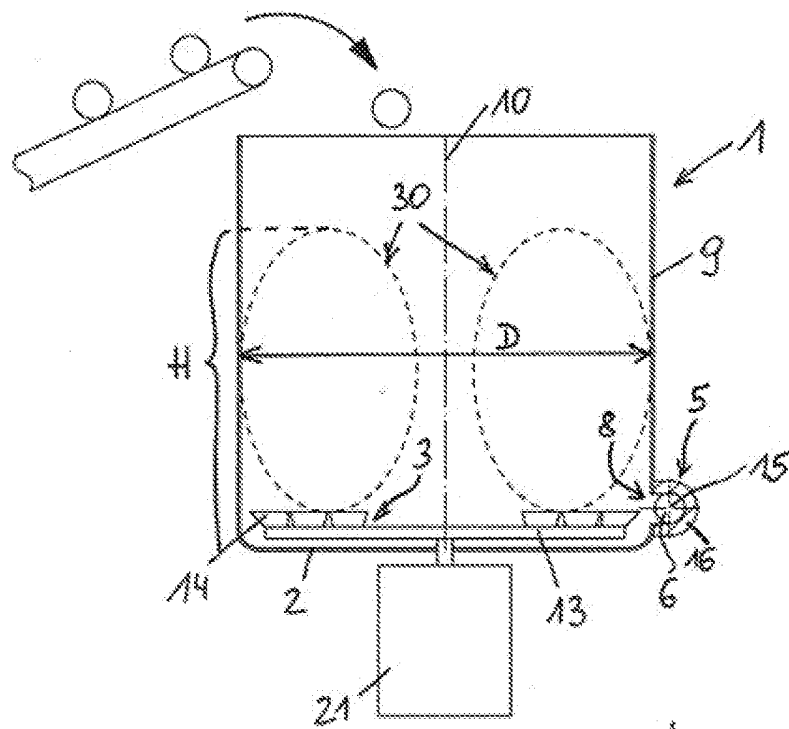
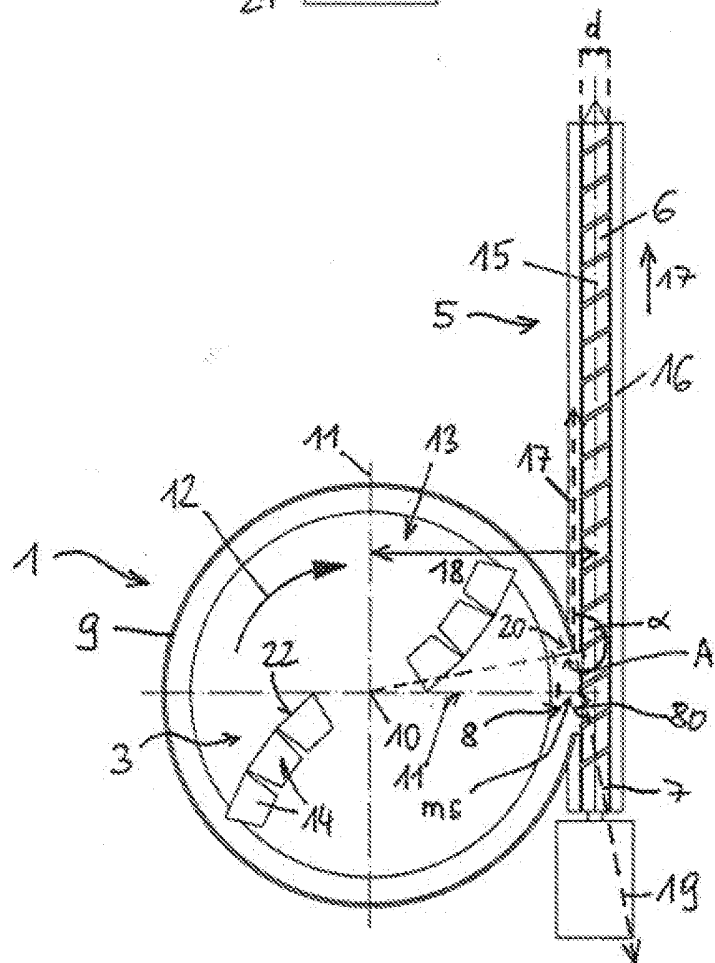


Fig. 2



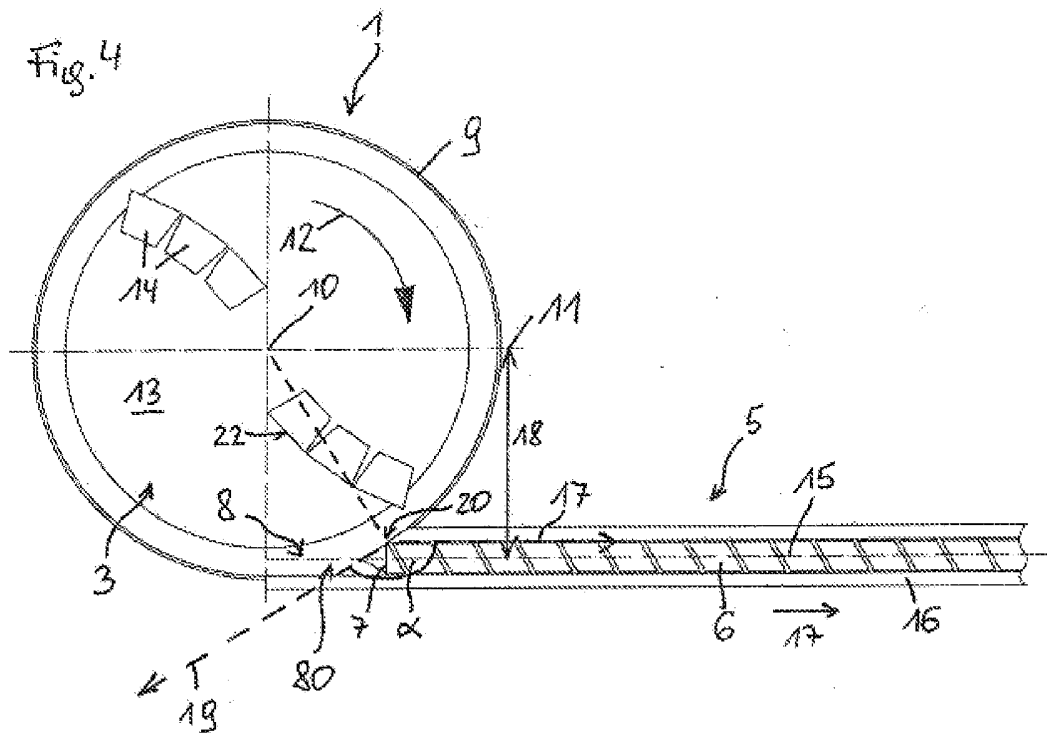
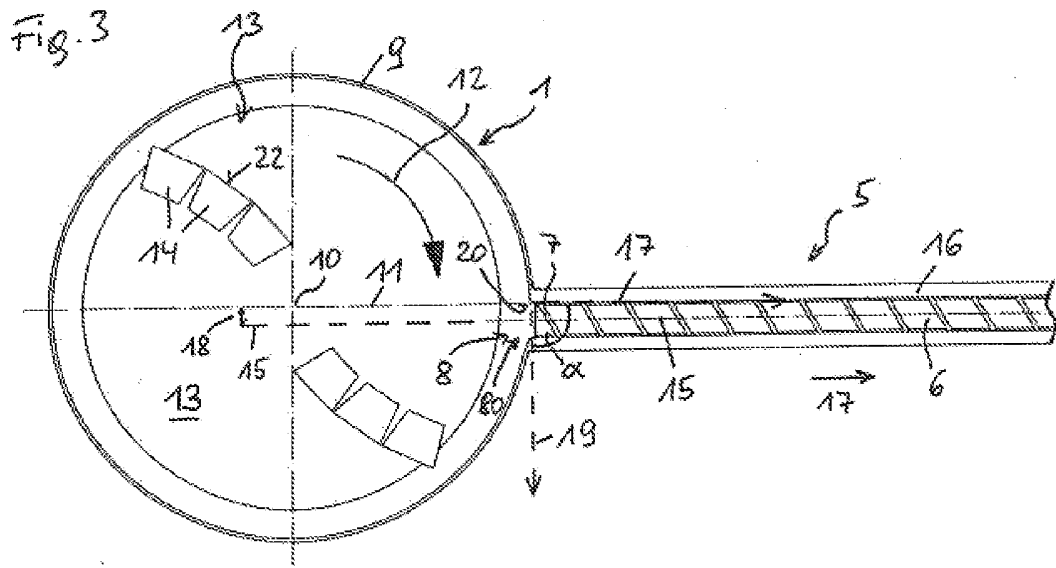


Fig. 5

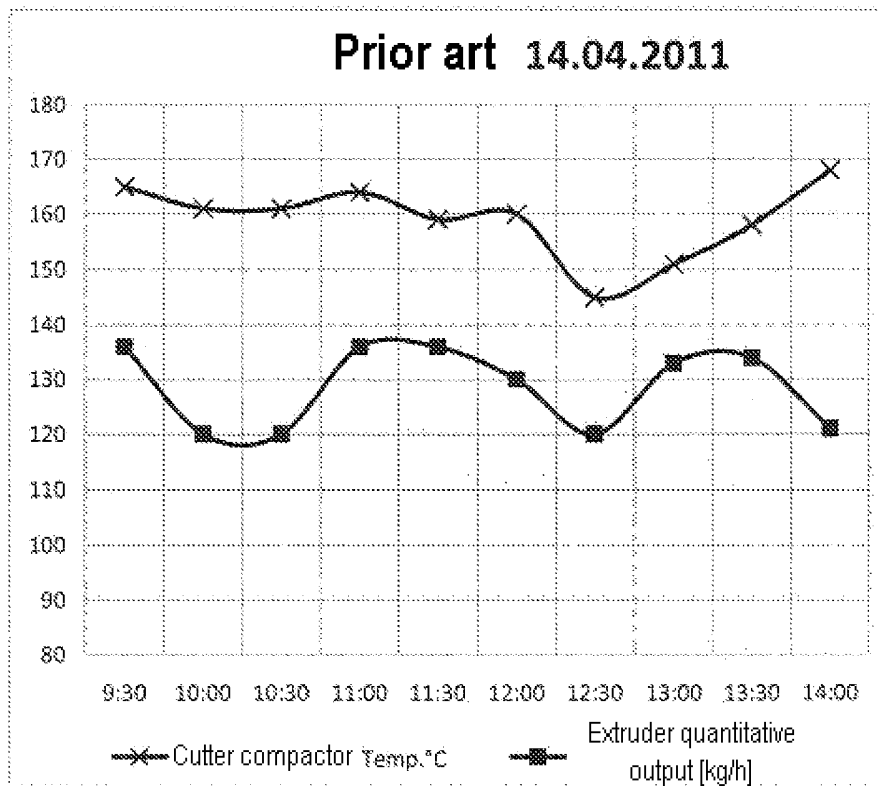


Fig.6

