

[54] ORBITAL AGITATING APPARATUS

[76] Inventor: Harvey C. Hoyt, 22744 Alcalde, Cupertino, Calif. 95014

[22] Filed: Nov. 19, 1973

[21] Appl. No.: 416,752

[52] U.S. Cl. .... 51/163; 51/164; 259/72

[51] Int. Cl. .... B24b 31/04

[58] Field of Search ..... 51/163, 164, 7; 259/72

[56] References Cited

UNITED STATES PATENTS

721,649	2/1903	Pendleton .....	51/164 UX
3,195,279	7/1965	Landon .....	51/164
3,390,490	7/1968	Hesslenberg .....	51/163
3,500,590	3/1970	Kergolan .....	51/164
3,609,921	10/1971	Foster .....	51/164

Primary Examiner—Harold D. Whitehead  
Attorney, Agent, or Firm—Thomas H. Olson

[57] ABSTRACT

A machine for agitating an article such as a container of small parts and an abrasive medium as employed in deburring or otherwise finishing the small parts. The container is rigid with a platform having a circular periphery and is driven by a flexible torque transmission element secured centrally of the circular periphery. The periphery rests on a table that has a central opening for the torque transmission element, and the point of contact between the circular periphery and the table makes an orbital path around the opening. The height of the table is adjustable in order to vary the configuration of the orbital path. The torque transmission has an eccentric element whereby the orbital path can be adjusted to be eccentric of the opening thus affording variation of the degree of agitation to which the container and contents are subjected.

13 Claims, 10 Drawing Figures

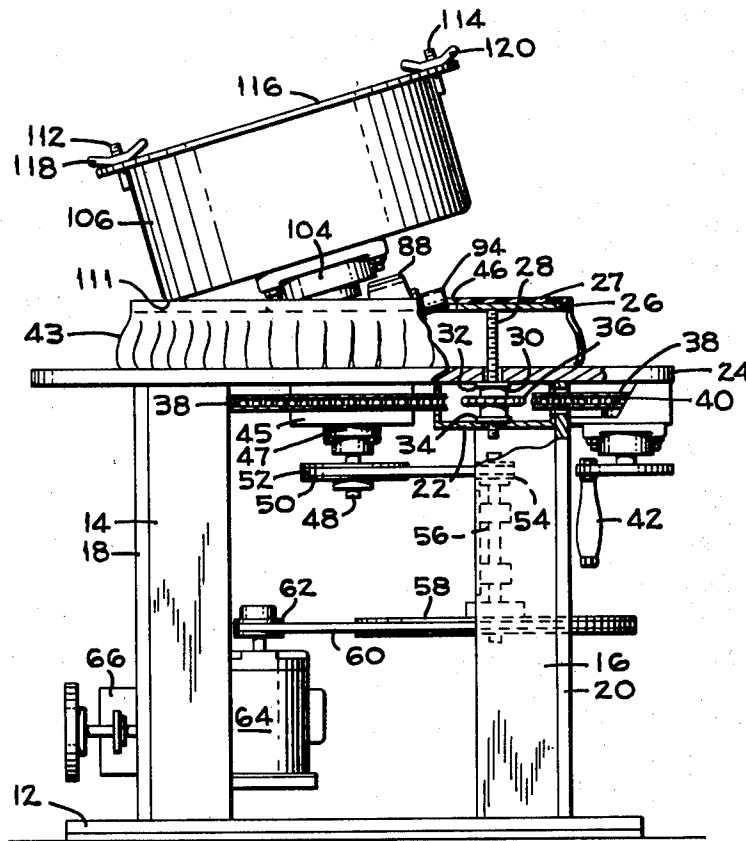
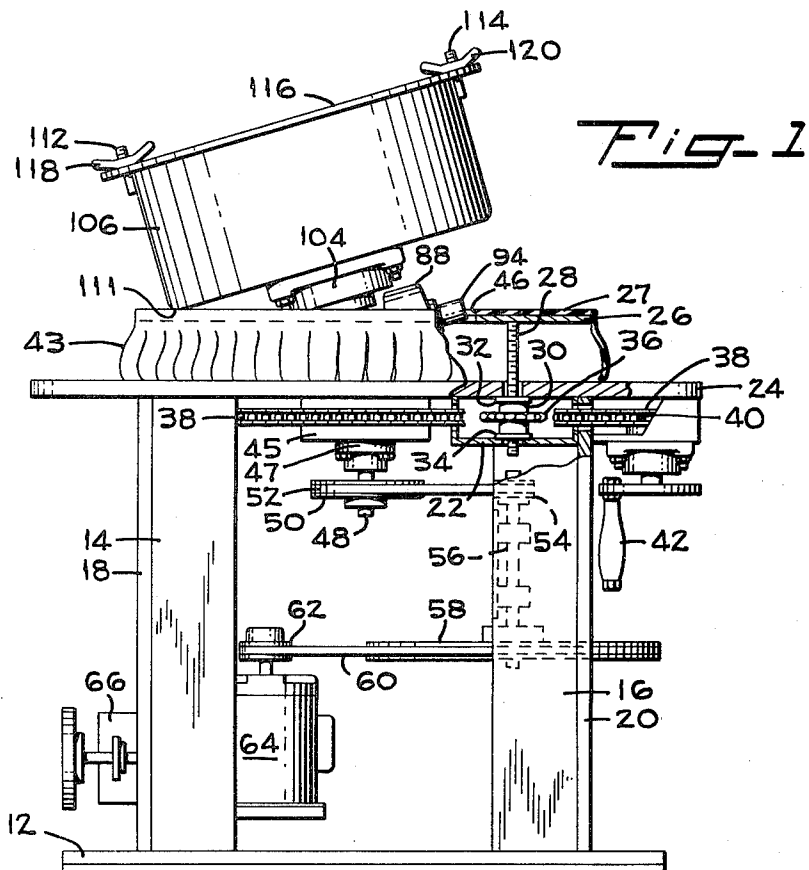
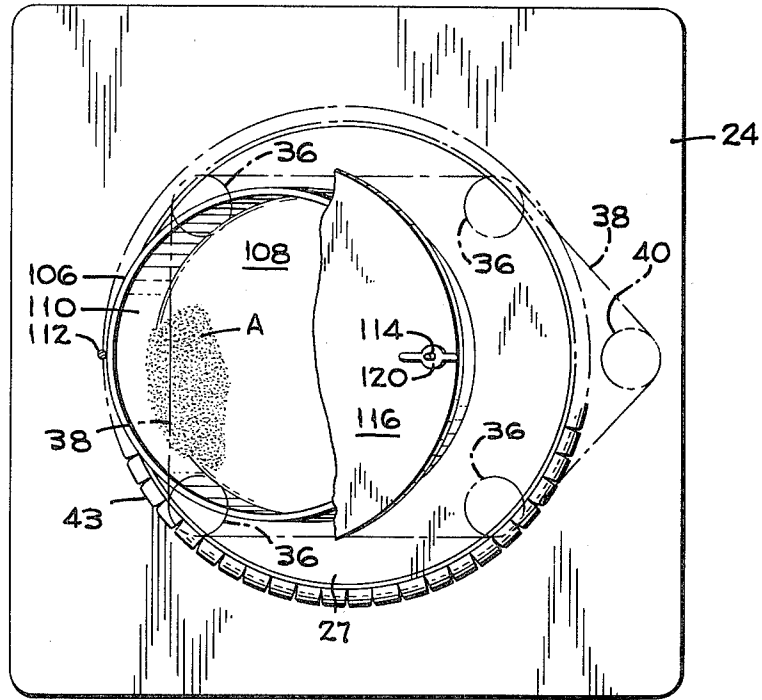
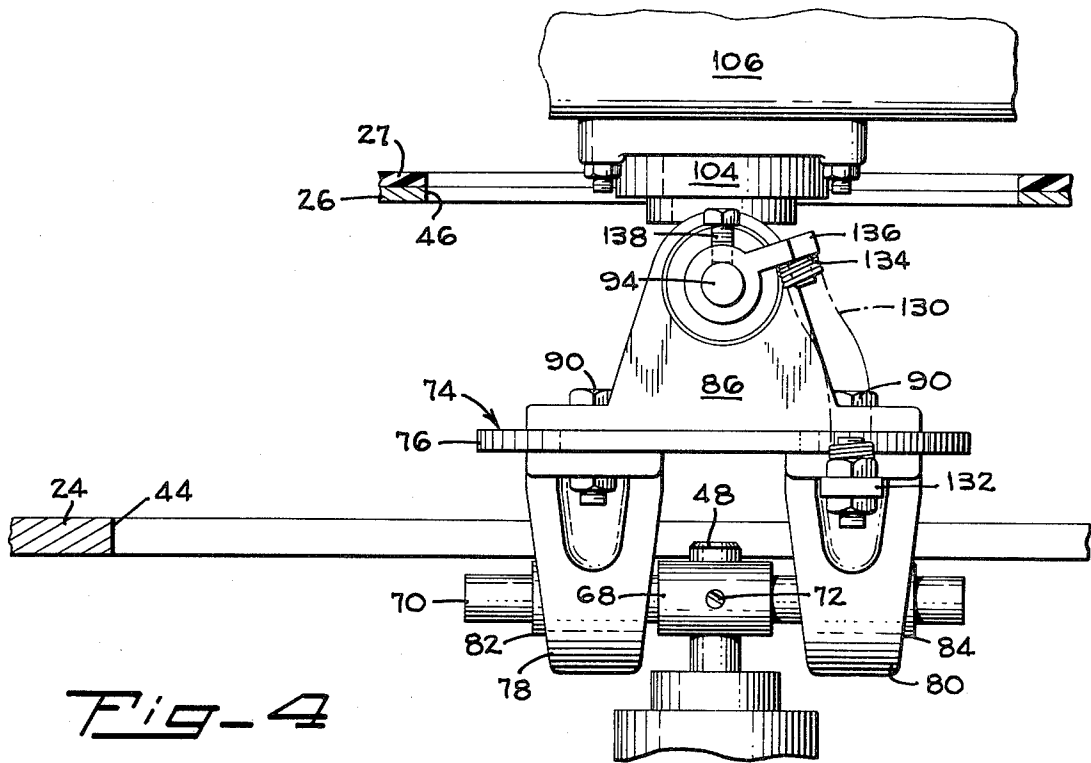
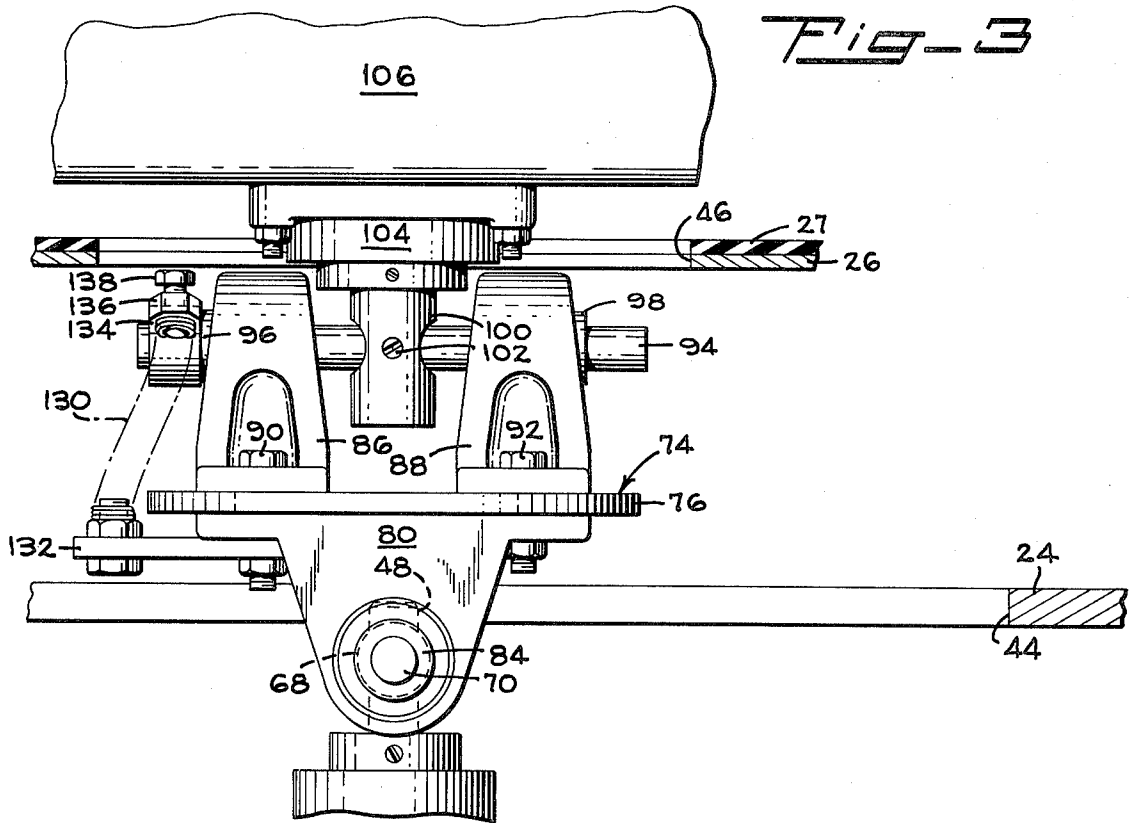
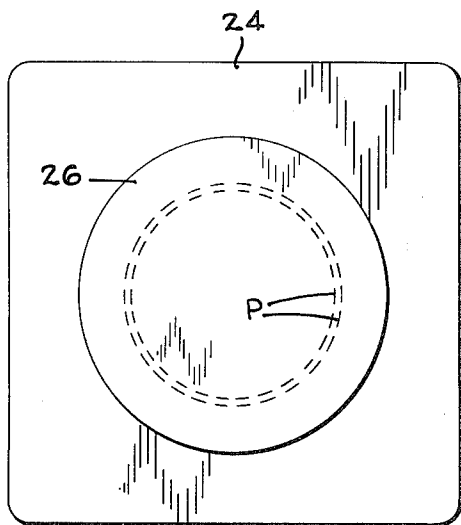


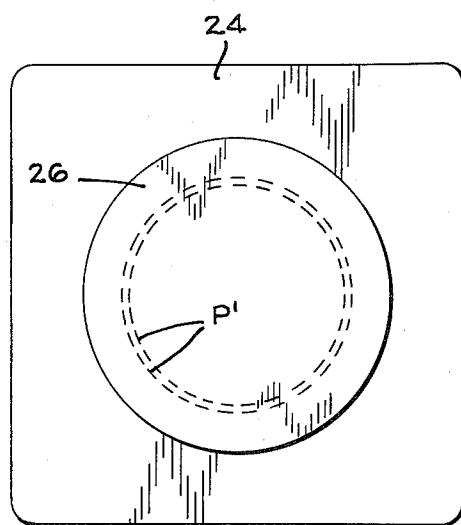
Fig-2





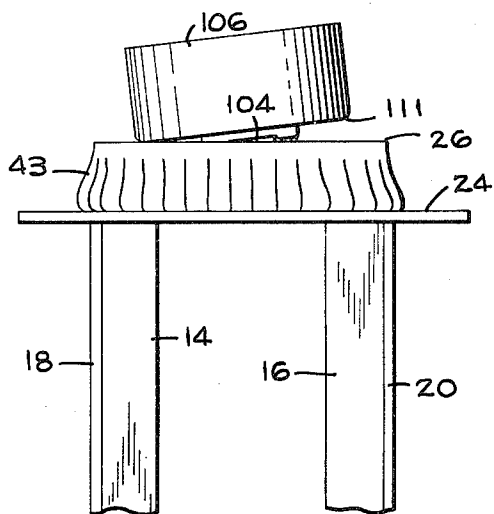


*Fig-5A*



*Fig-6A*

*Fig-5*



*Fig-6*

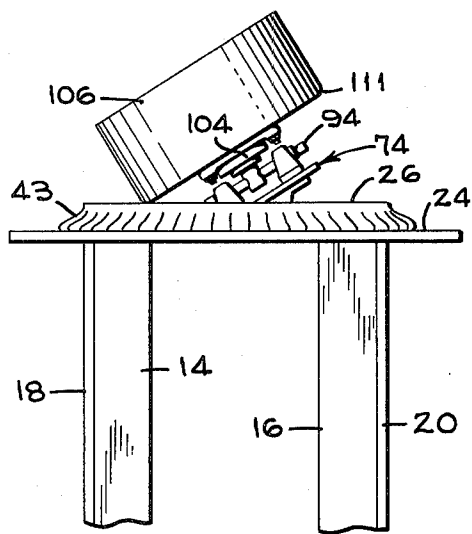


Fig-7

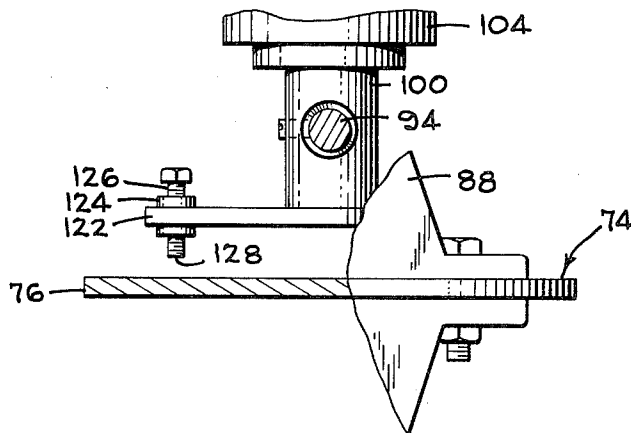
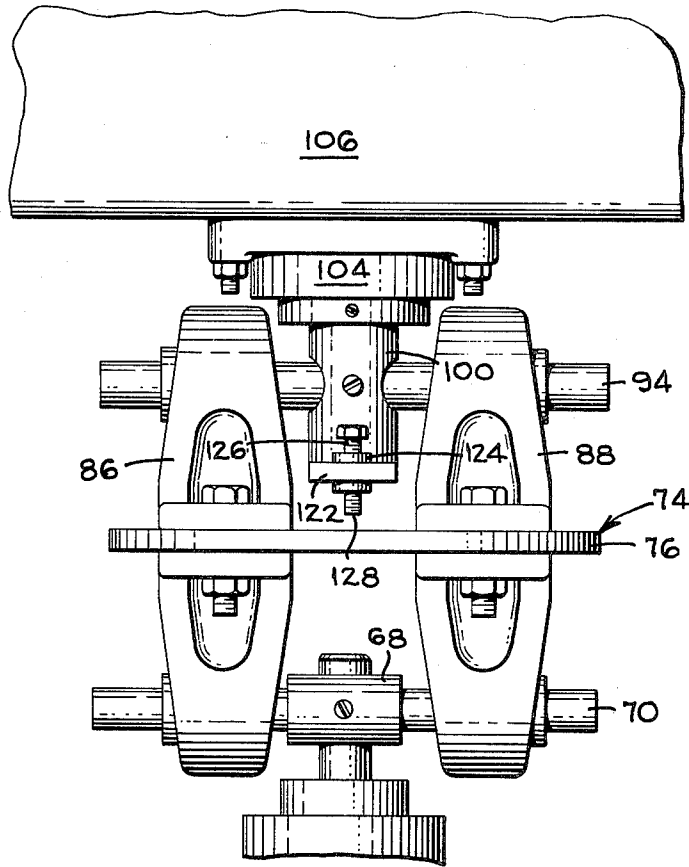


Fig-7A

## ORBITAL AGITATING APPARATUS

### FIELD OF THE INVENTION

The invention relates to a machine for agitating an article, and more particularly, to a machine for agitating a container adapted to receive a plurality of small parts and an abrasive or like surface finishing medium. The machine is exemplified hereinafter by a finishing machine for small parts.

### DESCRIPTION OF THE PRIOR ART

Known existing finishing machines include those employing a cylindrical barrel which is simultaneously rotated about two or more axes and finishing machines that are vibratorily driven to effect relative movement between the parts and the finishing medium therein. Although such prior art machines have been used satisfactorily, they tend to be overly complex and expensive and under certain conditions do not effect efficient treatment of the parts placed therein.

### SUMMARY OF THE INVENTION

The present invention subjects the article mounted to it, e.g., a container for parts and finishing medium, to a compound rocking orbital motion which effects efficient movement of the parts relative the finishing medium without subjecting the parts to unnecessary impact.

An object of the invention is to provide a device of the type referred to above in which the degree of agitation to which the parts are subjected is adjustable. Adjustability permits the machine to be operated at an optimum mode so that the parts are not subjected to undue impact but are moved with sufficient vigor that the requisite surface finishing is achieved. Adjustability of the degree of agitation is achieved in two ways, one by the speed of rotation of the orbiting container and secondly, by the angle through which the container is rocked in traversing the orbital path.

Another object is to provide a machine of the type referred to above that subjects the article to forces and/or motions in a plurality of directions. This object is achieved because the present invention agitates the article in a compound mode that includes rotation, which produces centrifugal forces, and rocking motion which produces oscillatory forces on the article. It has been found that a machine according to the present invention has a much higher production rate than prior art machines of equivalent power rating.

The foregoing, together with other objects, features and advantages will be more apparent after referring to the following specification and accompanying drawings.

### BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a front elevation view of an agitating machine according to the present invention.

FIG. 2 is a top view of the machine in FIG. 1.

FIG. 3 is a front elevation view of one satisfactory form of flexible torque transmitting structure according to the present invention, the structure being shown in an upright position for clarity.

FIG. 4 is a side elevation view of FIG. 3.

FIG. 5 is a partially diagrammatic front elevation of the device in one mode of operation.

FIG. 5A is a top view of FIG. 5.

FIG. 6 is a partially diagrammatic front elevation of the device in another mode of operation.

FIG. 6A is a top view of FIG. 6.

FIG. 7 is a front elevation view of an alternate form of a torque transmitting element according to the present invention.

FIG. 7A is a fragmentary side view of FIG. 7.

### DESCRIPTION OF THE PREFERRED EMBODIMENTS

Referring more particularly to the drawing, reference numeral 12 indicates a lower base plate on which the frame for the machine of the invention is supported. Extending upward from base plate 12 are four rigid legs, two of which are indicated in FIG. 1 at 14 and 16. One or more cover panels such as indicated at 18 and 20 in FIG. 1 can be installed in spanning relation of the legs in order to enclose the volume circumscribed by the legs. Mounted on top of legs 14 and 16 and their counterparts at the rear of the machine is a top 24 which is secured, such as by welding to the top of the legs. Overlying table 24 in substantial parallel relation thereto is a plate 26 which defines a generally horizontal upper surface. Plate 26 has a high friction layer thereon, such as a layer of rubber 27.

Plate 26 is arranged for vertical movement toward and away from table 24. To achieve this mode of operation there are secured to the lower surface of plate 26 four screws 28 (only one being shown in FIG. 1) which are arranged in a square pattern on the lower surface of plate 26 in alignment with legs 14 and 16 and their counterparts at the rear of the machine. With reference to FIG. 1, each screw 28 is threadedly engaged by a collar 30 which is captured between the lower surface of table 24 and the web of U-shaped member 22. Thrust washers 32 and 34 confine collar 30 against vertical movement and afford rotation of the same. Rigid with collar 30 and secured exterior thereto is a sprocket 36 which is engaged by a roller chain 38. Roller chain 38 engages each of the four sprockets associated with each of the four screws 28. Thus movement of roller chain 38 effects simultaneous rotation of all collars 30 and effects corresponding vertical movement of screw 28 and plate 26. Exterior of plate 20 is a drive sprocket 40 that is engaged with roller chain 38; drive sprocket 40 is driven by a hand crank 42 so that upon rotation of the hand crank, plate 26 moves up or down with respect to table 24. A flexible rubber skirt 43 is provided between table 24 and plate 26 as a safety measure to prevent access to the space therebetween.

Table 24 and plate 26 define central openings 44 and 46, respectively. Aligned with the openings and journaled for rotation in a bearing 47, which is fixed to a bearing support 45 on the base, is a shaft 48. On the lower end of shaft 48 is a sheave 50 which through a V-belt 52 and a sheave 54 receives power from a countershaft 56. On the lower end of countershaft 56 is a large sheave 58 which is driven through a V-belt 60 from a motor pulley 62 of a motor 64. A speed controller 66 is provided for controlling the speed of rotation of motor 64 and therefore of shaft 48.

At the upper end of shaft 48 is a collar 68 which joins shaft 48 to a pivot shaft 70. A set screw 72 fixes collar 68 to pivot shaft 70.

Pivot shaft 70 is carried in a rigid body 74 that forms a part of the flexible torque transmission means of the present invention. Body 74 includes a plate 76 to which

are rigidly connected bearing housings 78 and 80 which at their lower extremities support pivot shaft 70 for pivotal movement relative to the bearing housings. Collars 82 and 84 engage pivot shaft 70 to inhibit axial movement of the pivot shaft; the collars are adjustable, however, along pivot shaft 70 so that shaft 48 and collar 68 can be positioned symmetrical of bearing housings 78 and 80 or asymmetrical thereof.

To the upper surface of plate 76 are secured similar bearing housing members 86 and 88, bolts 90 and 92, serving to secure both the upper and lower pairs of bearing housings to plate 76. At the outer extremities of bearing members 86 and 88 is supported for rotation relative to the bearing members an upper pivot shaft 94, collars 96 and 98 being secured to the shaft to inhibit axial movement thereof. Collars 96 and 98 are identical to collars 82 and 84 in that they are adjustable in order that the longitudinal midpoint of pivot shaft can be positioned symmetrically or asymmetrically of bearing members 86 and 88. Secured to the midpoint of pivot shaft 94 is a shaft 100, there being a set screw 102 which engages pivot shaft 94 to retain shaft 100 to the midpoint of the pivot shaft. The upper end of shaft 102 terminates in a bearing assembly 104, one portion of which is secured to the article to be agitated, here exemplified by a cylindrical container or tub 106. Bearing assembly 104 affords rotation of container 106 relative to shaft 100, but inhibits axial and tilting movement of the container relative to the shaft. The cylindrical container includes a circular bottom wall 108 and a cylindrical side wall 110. Bottom wall 108 can be thought of as a platform having a circular rim 111, a portion of which is in contact with the upper surface of layer 27 on plate 26. Fixed to the upper margin of cylindrical wall 110 is a pair of threaded studs 112 and 114. A cover plate 116 is provided, there being holes for registry with threaded studs 112 and 114 in order that the cover plate can be installed to close the container. The cover plate is maintained in position by wing nuts 118 and 120 threaded on threaded studs 112 and 114, respectively. Container 106 and cover plate 116 are typically lined with rubber to avoid damaging the parts under treatment and to enhance transmission of motion to the finishing medium within the container.

The operation of the apparatus of the invention is as follows:

A plurality of small parts and finishing compound are placed in container 106 as indicated at A in FIG. 2. Cover plate 116 is placed onto the container and wing nuts 118 and 120 are engaged with the protruding portions of threaded studs 112 and 114 to secure the cover in place. The weight of container 106 and its contents causes the container to tilt so that the lower circular rim 111 of bottom wall 108 engages the upper surface of plate 26 at a limited peripheral region of the bottom wall or lower rim of the container. The flexible torque transmission affords this condition by pivotal movement of body 74 about the axis of pivot shaft 70 and/or pivotal movement of pivot shaft 94 with respect to the body. Motor 64 is then activated so that shaft 48 and body 74 of the flexible torque transmission means are rotatively driven. In response to such rotation shaft 100 is rotated and in turn the lower rim 111 of container 106 in contact with plate 26 rolls on the upper surface of the plate to describe an orbital path, such path being indicated in FIG. 5A by reference character P. The friction afforded by layer 27 causes the container to

roll but the force of friction is not so great as to inhibit sliding or slipping movement of rim 111 on layer 27, such as occurs when the vertical position of plate 26 is effected. Accordingly, container 106 is rolled around the surface of plate 26 and is subjected to both orbital and tilting oscillatory motion. The circumference of rim 111 of container 106 is different from a sub-multiple of the length of the path P on plate 26 over which the rim rolls, in consequence of which the container rotates relative to drive shaft 48. More specifically, as viewed in FIG. 5, clockwise rotation of shaft 48 causes counter clockwise rotative motion of container 106. Such motion is transmitted to the contents A of the container so that the action necessary for surface finishing is effected. In FIG. 5A it will be noted that the path P is represented by two broken lines. The two broken lines indicate the extremities of the orbital path. The radial distance between the two extremities is determined by the eccentricity of collar 68 on pivot shaft 70 and/or the eccentricity of shaft 100 on pivot shaft 94.

Comparison of FIGS. 5A and 6A indicates that the mean diameter of path P is determined by the vertical position of plate 26. Thus when the plate is relatively high (see FIG. 5) container 106 is tilted to a slight degree and the diameter of the orbital path is relatively small. When, however, plate 26 is relatively low (see FIG. 6) the angle of tilt of container 106 is relatively steep and the diameter of the radial path P' is relatively large. Moreover, the rate of reverse rotation of container 106 relative to shaft 48 is less when plate 26 is relatively high (FIG. 5) than when the plate is relatively low (FIG. 6). Thus the degree of agitation to which container 106 and contents A are subjected can be adjusted by rotation of crank 42 to raise or lower plate 26. Through speed controller 66, the speed of motor 64 and therefore of shaft 48 can also be adjusted. Thus the invention affords a wide degree of adjustment of agitation forces and speed according to the nature, weight and size of the article being agitated. Because both crank 42 and speed controller 66 can be adjusted while container 106 is rotating, it is possible to adjust or "tune" the degree of agitation in accordance with observation of the motion of the article.

The weight of container 106 and the contents A thereof is ordinarily sufficient to maintain a portion of rim 111 in contact with layer 27 on plate 26 and to assure adequate friction therebetween. Additional force can be produced, however, by a compression spring 130. The lower end of compression spring 130 is fixed to plate 76 by a connection 132. The upper end of the spring is secured at 134 to a lever arm 136 which is clamped around pivot shaft 94 by a clamping screw 138. The force of spring 130 biases pivot shaft 94 in a counter clockwise direction as viewed in FIG. 4 so as to resiliently urge a region of rim 111 into contact with layer 27 on plate 26.

FIG. 7 shows a modification of the flexible torque transmission according to the present invention. The individual parts of FIG. 8 are substantially identical to those described in connection with FIGS. 4 and 5, the principal difference being that bearing housings 86 and 88 are installed so that pivot shafts 94 and 70 are parallel to one another. In order to maintain container 106 in a tilted position so that only a limited peripheral region of rim 111 is in contact with the surface of layer 27 on plate 26, there is a rigid arm 122 secured to the

lower end of shaft 100 and extending outward from the central axis thereof. At the outer end of arm 122 is an anteriorly threaded bushing 124 in which is threadedly engaged a bolt 126. The lower end of bolt 126, indicated at 128, bears against the upper surface of plate 76 to limit the angular movement of shaft 100 on pivot shaft 90 so as to maintain container 106 in the requisite tilted position.

The operation of the embodiment of the invention of FIG. 8 is substantially identical to that described hereinabove in that actuation of motor 64 effects rotation of shaft 48 and body 74. Body 74 tilts so that a limited peripheral region of rim 111 of container 106 contacts the upper surface of plate 26. Further downward movement of the container is inhibited when bearing surface 128 of screw 126 contacts the upper surface of plate 76. Container 106 is thus rolled around the surface of plate 26 in an orbital path, the nature of the path being controllable by the vertical adjustment of plate 26 and by the degree of eccentricity of collar 68 along pivot shaft 70.

Although the invention has been described hereinabove in connection with a surface finishing machine, it is to be understood that the invention is useful for agitating virtually any type article. For example, a clamping mechanism for a paint can can be installed on bearing assembly 104 so that the paint can be readily mixed by use of the invention. Additionally, the invention may be employed in gravity separation process such as separating relatively dense gold from relatively light weight sand. These examples are not exhaustive of the nature of articles that can be agitated according to the present invention.

Thus it will be seen that the present invention provides an agitating machine which subjects the article to agitation in several directions or motions. This is achieved by a structure that is relatively simple and that can be adjusted over a wide range of degrees of agitation so that optimum results can be achieved.

Although two embodiments of the invention have been shown and described, it will be obvious that other adaptations and modifications can be made without departing from the true spirit and scope of the invention.

What is claimed is:

1. A machine for agitating an article comprising means defining a generally horizontal substantially planar non-rotatable surface, said surface defining means having an opening therein, flexible torque transmission means extending through said opening, an article supporting platform and means for attaching said platform to said torque transmission means above said surface, said attaching means including means for affording rotation of said platform relative said torque transmission means, said platform having a generally circular rim generally concentric with said rotation affording means, said flexible torque transmission means flexing to permit a peripheral region of said circular rim to bear on said surface, means below said surface for driv-

ing said torque transmission means so that said peripheral region of said circular rim describes a closed path on said surface around said opening, and means for securing the article to said platform.

2. A machine according to claim 1 including means for adjustably raising and lowering said surface defining means relative to said torque transmission means so as to vary the configuration of said closed path.

3. A machine according to claim 1 wherein said torque transmission means comprises a rigid body, upper and lower pivot shafts journaled in said body at spaced apart locations thereon, means for operatively connecting said driving means to said lower shaft so that said body is rotated by said driving means, and means for operatively connecting said rotation affording means to said upper shaft.

4. A machine according to claim 3 wherein said upper and lower shafts reside in respective upper and lower spaced apart planes.

5. A machine according to claim 4 wherein said upper and lower shafts are disposed so that their respective longitudinal axes are substantially perpendicular to one another.

6. A machine according to claim 4 wherein said upper and lower shafts are disposed so that their respective longitudinal axes are substantially parallel to one another.

7. A machine according to claim 3 wherein the axis of said lower pivot shaft is offset from the axis of said upper shaft so that said path is eccentric of said surface opening.

8. A machine according to claim 7 including means for adjusting the position of said lower shaft connecting means along the lower pivot shaft so as to afford adjustment of the degree of eccentricity of said path relative said opening.

9. A machine according to claim 3 including means for rotatively resiliently biasing said upper pivot shaft relative said rigid body to resiliently urge said peripheral region of said circular member into contact with said surface.

10. A machine according to claim 1 wherein said article securing means includes a rigid impervious container, said container having an access opening for affording introduction of parts and surface finishing material into said container.

11. A machine according to claim 10 in combination with a closure member for said opening and means for removably securing said closure member in sealing relation to said opening.

12. A machine according to claim 1 wherein said torque transmission driving means includes means for adjusting the rotative speed of said driving means to afford adjustment of the force of agitation on the article.

13. A machine according to claim 1 wherein said horizontal surface forming means includes a resilient high friction layer laminated thereto.

\* \* \* \* \*