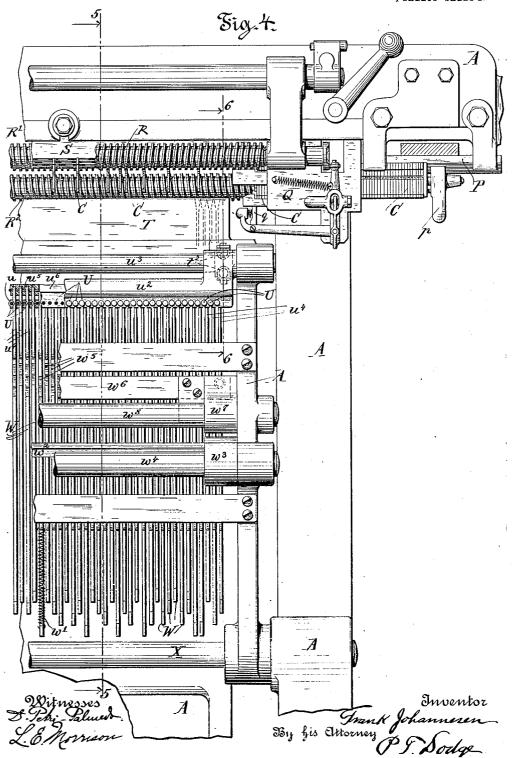
# F. JOHANNESEN. LINOTYPE MACHINE. APPLICATION FILED APE. 28, 1906.

4 SHEETS-SHEET 1.

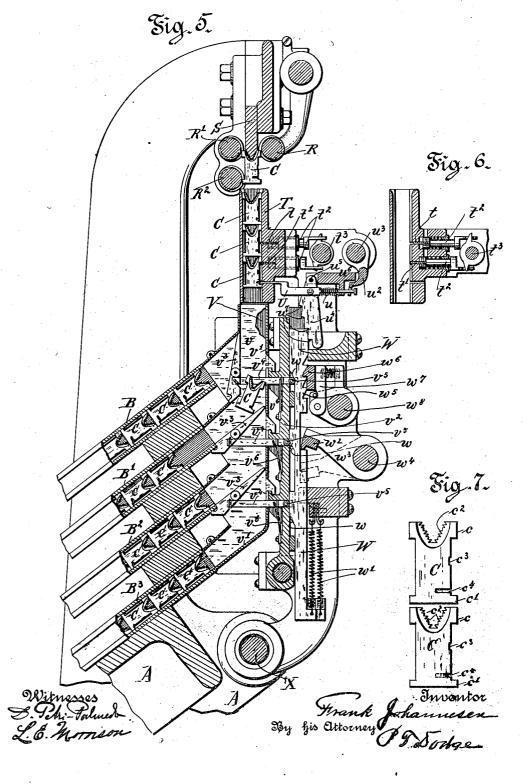
# F. JOHANNESEN. LINOTYPE MACHINE. APPLICATION FILED APR. 28, 1906.

4 SHEETS-SHEET 2.



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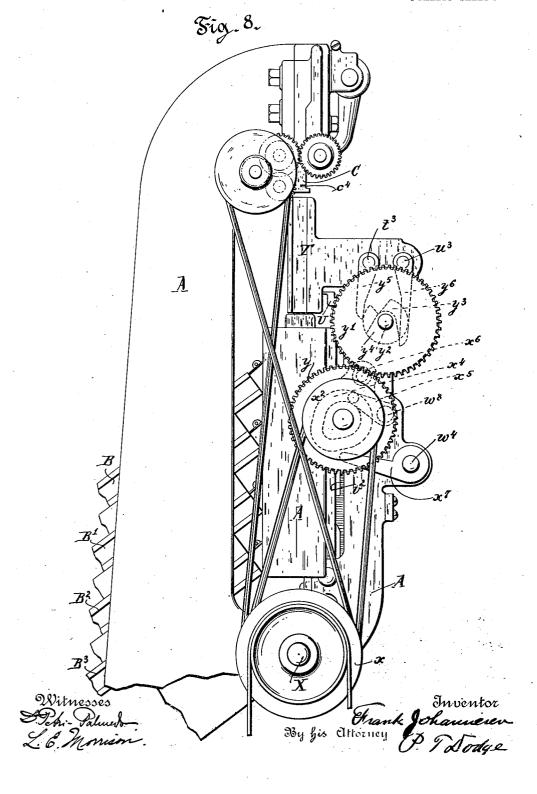
4 SHEETS-SHEET 3.



### F. JOHANNESEN. LINOTYPE MACHINE.

APPLICATION FILED APR. 28, 1906.

4 SHEETS-SHEET 4



### UNITED STATES PATENT OFFICE.

FRANK JOHANNESEN, OF ERIE, PENNSYLVANIA, ASSIGNOR TO MERGEN-THALER LINOTYPE COMPANY, A CORPORATION OF NEW YORK.

#### LINOTYPE-MACHINE.

No. 831,003.

Specification of Letters Patent.

Patented Sept. 11, 1906.

Application filed April 28, 1906. Serial No. 314,128.

To all whom it may concern:

Be it known that I, Frank Johannesen, of Erie, county of Erie, and State of Pennsylvania, have invented a new and useful Improvement in Linotype-Machines, of which

the following is a specification.

My invention has reference to linotypemachines of the general organization represented in Letters Patent of the United States 10 No. 436,532, wherein circulating matrices are released from a magazine in the order in which their characters are to appear in print and assembled in line, together with expanding spacers, the composed line transferred to 15 the face of a mold, the mold filled with molten metal to form a slug or linotype against the matrices which produce the type characters thereon, and the matrices thereafter elevated and returned through a dis-20 tributing mechanism to the magazine-channels from which they started.

The aim of my invention is to adapt a machine of this character to carry a number of fonts or sets of matrices, representing different sizes or styles of type, in such manner that any font may be brought into action at will, so that a line may be produced with faces represented by the matrices in any one magazine or by a combination of faces repre-30 sented by the matrices in two or more maga-

zines.

A further object of the invention is to provide for the distribution of the matrices from the composed lines to their appropriate 35 magazines without interrupting the composition of succeeding lines, and this whether the composed lines require to be distributed to one or more of the magazines. To this end I employ a series of magazines, preferably 40 superposed, in combination with movable means for directing the matrices from the respective magazines to a common assembling mechanism. I also employ means for preventing the discharge of matrices from 45 any but the one magazine in use at the time, as well as means by which continuous closed channels or chutes are formed for the passage of the matrices from the mouth of the magazine in use at the time to the assembling 50 mechanism. I furthermore employ a main distributer, by which the matrices of a line are first separated according to the letters of the alphabet and distributed into channels

of a common intermediate or auxiliary magazine and secondary or auxiliary distributing 55 means by which the matrices are further separated into fonts and distributed into the channels of the various individual principal magazines, a continuous closed channel being preferably provided for each matrix from 60 its channel of the auxiliary magazine to its appropriate channel in the individual magazine to which it belongs. I also employ for the purpose of effecting the separation into fonts matrices with distinguishing features 65 to cooperate with the distributing means.
In the accompanying drawings I have rep-

resented my improvements as incorporated in a machine of the same general organization as the commercial Mergenthaler lino- 70 type-machine of the present day, and I have restricted the drawings to those parts of the machine which are necessary to an understanding of my improvements; but it is to be understood that the improvements may be 75 employed in connection with any other suitable line-casting machine and that the details may be widely modified within the range of mechanical skill and without passing beyond the limits of my invention.

Referring to the drawings, Figure 1 is a front elevation of a portion of the assembling mechanism, showing the magazines and what is commonly called the "assembler-entrance," the escapements, and assembler-belt all lo- 85 cated at the lower or discharge end of the magazines. Fig. 2 is a vertical section from front to rear of this portion of the machine on Fig. 3 is a detail perspective line 2 2, Fig. 1. view of two of the escapements. Fig. 4 is a 90 rear elevation of the upper or receiving portion of the magazines, showing the distributing means. Fig. 5 is a vertical section from front to rear of this portion of the machine on line 5 5, Fig. 3. Fig. 6 is a detail section on 95 line 6 6, Fig. 3. Fig. 7 is a view of two of the matrices employed in the machine. Fig. 8 is a side elevation of the distributing or top portion of the machine, illustrating the mechanical connections of the various moving 100 parts of the distributing mechanism.

Referring to the drawings, A represents a rigid main frame, which may be of any form and construction adapted to sustain the various operative parts.

B B' B2 B3 are inclined magazines super-

imposed and rigidly connected to each other ! and the main frame, each magazine consisting of a pair of plates channeled longitudinally and separated by distance-pieces and 5 adapted to contain a font or set of matrices C.

D represents escapement-actuating slides connecting at their lower ends with the finger-key mechanism and engaging at their upper ends centrally-pivoted levers E, Figs. 2 10 and 3. The slides D are supported in the usual manner by stationary guides H, are depressed by corresponding finger-keys, and raised by means of springs d in the wellknown manner. By the reciprocating mo-15 tion of the slides D a rocking motion is im-

parted to the levers E.

F and F' are escapement-slides for releasing the matrices resting at their lower ends on the extremities of the centrally-pivoted 20 levers E, passing through the series of magazines B B' B<sup>2</sup> B<sup>3</sup>, and acted upon at their upper ends by light springs G and G'. The rocking motion of one of the levers E causes a reciprocating motion of the slides F and F', 25 the one being raised while the other is low-The slides F and F' pass through the magazines between the channels containing the matrices and are provided with side projections or lips f and f', extending into the matrix-channels and the path of the matrices engaging the ears of the same, as shown particularly in Fig. 3. The reciprocating motion of the slides F and F' releases one matrix at a time. It will be observed that the two 35 oppositely-moving escapement slides or bars of each pair coöperate in controlling the delivery of each matrix on which they act. It will also be observed that each pair of slides, extending through the series of magazines, 40 controls the delivery of matrices from all the magazines.

There being but one set of escapement-slides F and F' for all of the magazines B B' B<sup>2</sup> B<sup>3</sup>, each of the latter would deliver a ma-45 trix upon the actuation of one of the escapements. To prevent this, I provide in front of the mouths of the magazines rock-shafts I The cross-section of these rockshafts is such that in one position they will 50 obstruct the passages of the matrices, as shown in Fig. 2, in the instance of rock-shafts I, I', and I<sup>3</sup>, and in another position leave the passages free, as in the instance of rock-

shaft I2.

J is a channel-plate to receive matrices from all the magazines and guide them downward toward the assembling devices. divided by vertical partitions j and has a glass cover K. Lis a continually-moving as-60 sembler-belt, upon which the matrices fall and by which they are carried laterally to the assembling-elevator. These parts are well known in the art. The channels J are divided horizontally for a portion of their

width by plates j', forming extensions of the 65 lower plates of the magazines and overlying the rock-shafts I I', &c. To prevent the ears of the matrices from catching on the edges of the plates j' in their descent past them from an overlying magazine, I provide bridging 70 pieces or flaps  $j^2$ , one for each channel J and magazine except the top one. These flaps  $j^2$  are mounted on rock-shafts  $j^3$ , connected by pinions  $j^4$ ,  $j^5$ , and  $j^6$  with the corresponding rock-shafts I I', &c. The latter carry handles  $j^7$ . Upon turning one of these handles not only the corresponding rock-shaft I is turned, but also the corresponding rock-shaft  $j^3$ , geared thereto, thus opening or closing, respectively, the corresponding magazine 80 and simultaneously opening or closing the gap between two adjacent plates j'. Thus, as shown in Fig. 2, the matrices of magazine B2 are free upon being released by the escapements to move into the passages J, the ob- 85 structing part of the rock-shaft I<sup>2</sup> being turned upward and rock-shaft j3 being turned so as to swing the flaps  $j^2$  outward, while the other rock-shafts I I' I<sup>3</sup> are obstructing the mouths of their respective magazines, and the 90 flaps  $j^2$  of the lowermost magazine B<sup>3</sup> form smooth rear walls of the channel J in front of said magazine for the safe and unobstructed passage of matrices from magazine B2.

The springs G and G' are preferably mounted on shafts g and g', so that they may be turned up and out of the way when it is desired to remove the escapements F and F' from the machine for inspection or repair. To facilitate such removal of the escapements, I 100 provide on each side of the magazines a sliding bar M, guided in the main frame and having racks in their lower ends, into which mesh pinions m, mounted on a shaft m', journaled in the main frame. These sliding bars M are 105 connected at the top by a cross-bar  $m^2$ , carrying a guide-plate  $m^3$  for the escapement-slides F and F' and provided with grooves  $m^4$ . The escapement-bars F F' have notches m5, registering with grooves  $m^4$  of the cross-bar  $m^2$ . 110 By inserting rectangular rods into the grooves  $m^4$  and notches  $m^5$  the cross-bar  $m^2$  may be securely locked to all of the escapement-bars. The shaft m' has a crank  $m^6$ , the turning of which raises or lowers through pinion m slid- 115 ing bars  $\dot{M}$ , its cross-bar  $m^2$ , and all the escapement-bars F and F' locked thereto.

N is a cover protecting the escapement slides and springs from dust and possible in-

O represents bars lying across the magazines and having lips o entering the same through their top plates. These bars O are all connected to two upright sliding bars o', which are guided in the main frame and may 125 be lowered by means of an eccentric shaft  $o^2$ , journaled in the main frame. By lowering the bars O the lips o obstruct the passage of

120

100

the matrices and lock them in place when the escapement-slides are removed for any rea-

son whatever.

After the matrices have been assembled 5 either all from one of the various magazines or from two or all of them into a line the line is carried in front of the mold and justified. The slug is then cast and the line removed from the mold, the spacers are extracted, and to the matrices elevated to the top of the machine in the well-known manner described in Patent No. 436,532, preparatory to distribution. As the machine contains in the respective magazines several distinct fonts or 15 sets of matrices, it is necessary that they should be separated or distributed according to the font and also according to letter in order that they may be delivered each to its proper magazine and to the appropriate 20 channel therein. To secure this result, I first separate or assort the matrices according to the characters therein, regardless of font, delivering all matrices of the same letter to the same point, and thereafter I separ-25 ate each of these letter-groups according to font and conduct each matrix to the proper channel in its magazine. To this end all matrices in the machine containing a given letter or character are provided with the 30 same combination of distributing-teeth; but matrices of different fonts are provided with font-distinguishing notches or equivalent distinguishing features common to all matrices of the same font, regardless of letter, as 35 hereinafter more fully explained.

P, Fig. 4, is the matrix-carrying rail of the so-called "second" elevator, of usual con-

struction. p represents the usual pusher that transto fers the matrices C to the distributer-box Q, whence the matrices are lifted one by one by the finger q between the distributer-screws R R' R2, that guide them upon the toothed distributer-bar S, as in the ordinary Mergen-45 thaler linotype-machine described in United States Letters Patent No. 436,532. The matrices are, however, not directly distributed into their proper magazines, but into an intermediate upright magazine T, Figs. 4 and 30 5, constructed in the same manner as the ordinary magazines with grooves for the matrices, but shorter. Each of these grooves or channels receives all the matrices in the machine containing a given character—for ex-.55 ample, all the "A" matrices of the various fonts. In practice there will commonly be an accumulation of matrices in each groove belonging to one, two, or more fonts, but all containing the same character. At the back of this auxiliary magazine are mounted two blades t and  $\check{t}'$ , entering the magazine T through slots provided for the purpose. These blades engage the upper ears of the matrices and act as escapements. The blades

are fastened to spring-actuated plungers t2, 65 connected to a rock-shaft t3 in such a manner that when one blade—for instance, t— is out of the path of the matrices, the other, t', is in the path of the same, as shown particularly in Fig. 6. By rocking the shaft t3 one hori- 70 zontal row of matrices C, accumulated in the magazine T, will be delivered therefrom at a time.

The matrices C (see Fig. 7) are of the form used in the ordinary Mergenthaler linotype- 75 machine, with ears c and c', teeth  $c^2$  for effecting distribution according to letter, and intaglio characters c3, but have in addition each

a notch c\*, varying in depth according to fonts.

Through the rear plate and into the inte-80 rior of magazine T enters pawls U, one for each matrix-channel, registering with and entering the notches c<sup>4</sup> of the lowermost matrices, as shown in Fig. 5, while suspended on the blade t'. The pawls U are continually 85 urged in one direction by springs u, tending to push them into the magazine and into the notches c4 of the matrices and intermittently in the other direction by a rocking plate  $u^2$  on a shaft  $u^3$ , tending to withdraw them from 90 the magazine T. Each of the pawls U is connected by a pin to a swinging lever u4, pivoted at u5 to a stationary plate or support u6, forming part of the stationary framework of the magazine T. Each of the pivoted levers 95  $u^4$  has formed on its front edge steps  $u^7$ . According to the depth of the notch  $c^4$  in the matrix the pawl U will enter the magazine more or less, giving to the stepped pivoted lever  $u^4$  various positions.

Underneath the auxiliary magazine T a series of channels or throats V is formed by means of vertical partition-plates v and a back plate v', all fastened to a stationary Through these channels or throats 105 frame  $v^2$ . V the matrices are guided from the auxiliary magazine into the main magazines B B' B2 B3 over bridges or switches  $v^3$ , of which there is one for each matrix-channel of each magazine except the lowermost one. These bridges 110 or switches are pivotally mounted between the partition-plates v of the throats V and connected by pins to horizontally-sliding links  $v^4$ , guided in the back plate v' and the frame  $v^2$ . On their rear free ends the links  $v^4$ On their rear free ends the links v4 115 carry antifriction-rollers v5, arranged to ride upon inclined surfaces w of vertically-reciprocating slides W, the function of which will be explained presently. Proper contact of the rollers  $v^5$  with the inclined surfaces w is 120 assured by springs  $v^6$ , acting on the horizontal links  $v^4$ . The tendency of these springs is to pull the links backward and in turn swing the switches or bridges  $v^3$  into a position such as those belonging to magazines B' B<sup>2</sup> 125 B³ are shown to occupy in Fig. 5. In this position of the switches v3 they form continuations of the bottom plates of their respective

magazines, at the same time partitioning off ! the portions of the channels V below. In this position of the switches  $v^3$  their respective links v4 rest with their rollers v5 against 5 the narrow portions of the upright slides W. When pushed forward, the links  $v^4$  cause the switches  $v^3$  to swing up into a position such as that belonging to magazine B is shown to occupy in Fig. 5. From the figures it will be 10 clear that by the position of the various flaps a continuous passage-way to the matrices is formed from the channels of the auxiliary magazine T to those of the magazine B', the entrance to the top magazine B above being 15 closed, as well as that to the magazine B2 below. It will be clear also that by proper changes of the relative position of the switches v<sup>3</sup> any one of the main magazines may so be connected by closed channels with the auxil-20 iary magazine T to the exclusion of the others. This is effected automatically by means of the vertical slides W in conjunction with the stepped pivoted levers  $u^*$ . The slides W are constantly urged upward by springs w', at-25 tached to the bottom ends and the stationary frame, respectively, but are held down in their lowermost position by a cam-plate  $w^2$ , engaging in a recess of W and attached to arms  $w^3$  of a rock-shaft  $w^4$  until such time as 30 they are required to take different positions. It is to be noted that although the action of the switches  $v^3$  is controlled or determined by the individual matrices their movements are effected by power-driven connections. 35 insures speedy and certain action of the switches in advance of the arrival of the mat-The slides W are provided also on their rear edges with ratchet-teeth w, in number equal to the number of magazines 40 and the number of steps in the pivoted levers  $u^4$ . The ratchet-teeth  $w^5$  are engaged by a pawl-plate w<sup>6</sup>, carried on spring-actuated arms  $w^7$  of a rock-shaft  $w^8$ .

The various parts being described, the ac-45 tion of the mechanism is as follows: Matrix C having arrived in front of the pawl U and being held there by the escapement-blade t' for the time being, shaft u3 is rocked, releasing pawl U, leaving it free to follow the tend-50 ency of its spring u to enter the notch  $c^*$  of the matrix, as shown in Fig. 5. According to the depth of the notch  $c^4$ , pivoted lever  $u^4$  will swing forward more or less. In the figures it is shown to have swung so far as to 55 bring the second-lowest step into line with the top edge of the vertical slide W. accomplished, the shafts  $w^4$  and  $w^8$  are rocked upward and outward, respectively, leaving the slide W free to ascend as far as it may to under the influence of its spring w'. stopped in its upward motion by the step  $u^7$ of the lever  $u^4$ , presented in its path. shaft  $w^{s}$  is then rocked back, causing the pawlplate we to engage the proper ratchet-teeth

w<sup>5</sup> of the slide W, holding the same in posi- 65 tion against displacement during the actions that follow. This stage of the action of the mechanism is illustrated in Fig. 5, and the result accomplished thus far is that the slide W has risen to such an extent that roller  $v^5$  of 70 the uppermost link  $v^4$  has passed from the higher to the lower level of the inclined surface w, while the rollers of the other links are still on the higher levels. This causes the switches  $v^3$  to take the position shown in Fig. 75 5 and the formation of the passage-way, as previously described, to one magazine only. Immediately after the flaps have been set for the reception of the matrix into its proper magazine pawl U is withdrawn by rocking 80 the shaft  $u^3$  backward. The flaps are not affected by this, although the stepped lever u4 is also withdrawn from the top of the slide W, because the latter is still held in position by the pawl-plate  $w^6$ . Next the rock-shaft 85 is rocked and the escapement-blade t'caused to recede and the blade t to advance, whereupon the matrix C drops from the auxiliary magazine T into the channel V, and thence passes into magazine B', while the 90 next following matrix is held in suspension by the escapement-blade t until the shaft  $t^3$  is rocked back again, which occurs immediately afterward when this matrix assumes the position of its predecessor. Next the rock-shaft 95  $w^*$  is rocked back again, depressing the slide W to its extreme lower position, whereby all the switches  $v^3$  are swung down, all the antifriction-rollers  $v^5$  riding on the high levels of the slide. The cycle is now repeated by the rocking of shaft  $u^3$  and the advance of pawl U for the next matrix, &c.

It will be understood that although in describing the action of the distributing mechanism only one slide W, only one pawl U, &c., 105 has been named all the slides W and all the pawls U, as well as their contiguous parts, perform like motions simultaneously; also, that the lowest matrix in one channel of the auxiliary magazine may be of a different font 110 from that in the next channel and of a different font again from that in the third, &c., so that the three corresponding pawls U will advance different distances and cause through the intermediate mechanism a dif- 115 ferent disposition of the switches  $v^3$  in the different channels V. It will be further understood that in case any one channel of the auxiliary magazine is empty the action of the rock-shaft  $w^4$  will have no effect on the par- 120 ticular slide W belonging to that channel, being held in its lowermost position by its stepped lever  $u^4$ , and consequently the corresponding switches  $v^3$  will remain at rest until the channel receives a matrix. Since at all 125 times there will be a number of empty channels in the auxiliary magazine, the fact that only those mechanisms belonging to channels

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time reduces the number of moving parts to

The various rock-shafts  $t^3$ ,  $u^3$ ,  $w^4$ , and  $w^8$ 5 receive their motion from a shaft X, Figs. 4, 5, and 8, that also supplies power to the distributer-screws. From a pulley x power is transmitted by a belt x' to a pulley  $x^2$ , mounted on a stud  $x^3$ . Integral with the pulley  $x^2$  is a cam  $x^4$ , acting on an arm  $x^5$ , carried on rock-shaft  $w^8$ . The cam  $x^4$  carries a roller  $x^6$ , that acts at the proper moment on an arm  $x^7$ , carried on rock-shaft  $w^4$ . Integral with pulley  $x^2$  is a pinion y, meshing with another pinion y' on a stud  $y^2$ . Integral with the latter pinion y' are cams  $y^3$  and  $y^4$ , acting on arms  $y^5$  and  $y^6$ , carried on rock-shafts  $t^3$ and  $u^3$ , respectively.

All the matrix-fonts used in the machine 20 will contain usually the same letters and other characters; but the characters of one font will differ in size or style from those of the other fonts. All matrices containing the same character will have the same arrange-25 ment of distributing-teeth and will be delivered regardless of font from the distributerbar S at the same point and into one channel in the supplemental magazine T. Consequently the magazine T will hold momen-30 tarily for further distribution letters of different fonts assorted according to their characters, all "A's" in one channel, all "B's" in

another, &c.

The slides or blades t t' serve merely as es-35 capements to detain the assorted matrices momentarily until each one causes the setting of the devices which control its course into the proper magazine. It is to be noted that the devices which control the final 40 course of the individual matrices is powerdriven and positive in action. The action is dependent on the form of the matrices—that is to say, on the depth of their notches  $c^4$ .

It will be observed that the auxiliary 45 magazine T and the parts below it present continuous vertical intermediate channels or passages through which the matrices separated by the first distributer S are conducted to corresponding channels in the main maga-50 zines, all matrices containing the same character descending through the same channel. It will also be seen that the pivoted members  $v^3$ are, in effect, switches or guides to deflect the matrices from their vertical course sooner or 55 later, according to font, that they may enter one magazine or another.

I believe the present mechanism to be the first in which two or more fonts are first separated according to letter and power-driven 60 devices thereafter caused to determine the course of each matrix to one magazine or an-

other according to font.

I am aware that several fonts of matrices in a machine have been combined with means

having matrices in them are operative at the ; for separating them according to font and 65 with a series of distributing mechanisms each adapted to distribute the matrices of one font according to letter; but the arrangement herein shown and claimed is advantageous, in that it enables a single distributer to first 70 separate all the matrices according to letter, after which these letter groups may be separated according to font by very simple mechanism.

**5** . . .

Having described my invention, what I 75

1. In a linotype-machine, plural-channeled magazines in combination with sliding coöperating escapement-bars arranged in pairs and each extended through all the 80 magazines.

2. In a linotype-machine, parallel magazines containing matrix-guiding channels, in combination with cooperating escapementbars arranged in pairs and extended through 85 all the magazines, and means for reciprocating the bars of each pair in reverse directions.

3. In combination, plural parallel magazines, reversely-reciprocating cooperating es- 90 capement-bars arranged in pairs and extended through all the magazines, and means controllable at will for holding back the matrices of all but one magazine, whereby one series of escapements is enabled to deliver matrices 95 to any one of the magazines as demanded.

4. In combination, plural magazines, escapement-slides passing through the series of magazines and means for withdrawing the

slides in series from the magazines.

5. In combination with a series of escapement-slides, the bar m2 adapted for connection with the series of slides and mechanism for lifting said bar to withdraw the slides from the magazines.

6. In a linotype-machine, plural inclined magazines overlying one another and escapements to release the matrices therefrom, in combination with a channel-plate arranged to receive matrices from all the magazines, 110 and intermediate movable flaps to cover the ends of the inoperative magazines and assist in guiding the matrices past the same to the channel-plate.

7. In a linotype-machine, plural inclined 115 magazines and escapements to release matrices therefrom, in combination with a channel-plate arranged to receive matrices from all the magazines, and intermediate hinged

flaps  $j^2$ .
8. In a line type-machine, a series of superposed inclined magazines, a channel-plate arranged to receive matrices therefrom, in combination with rock-shafts I3 to retain matrices in the inoperative magazines, and 125 hinged flaps  $j^2$  to aid in guiding the released matrices.

9. In a linotype-machine, a series of paral-

lel magazines, a stop device for each magazine distinct from the escapement mechanism to hold the matrices therein, and means for operating said stop devices in series.

10. In a linotype-machine, plural magazines, escapement devices to control the delivery of matrices therefrom, and an independent mechanism adapted to lock the matrices in all the magazines by a single action.

11. In a linotype-machine, a series of magazines, a series of matrix-holding devices O, one for each magazine, connections between said stop devices, and means for oper-

ating them in unison.

12. In a linotype-machine, two inclined 15 magazines, one overlying the other; a channel-plate J arranged to receive matrices from both magazines, and an intermediate movable plate  $j^2$ .

13. In a linotype-machine, a series of inclined superposed magazines, a channel-plate J common to the series, a swinging plate j2 movable to and from the ends of the

magazines.

14. In a linotype-machine, a series of magazines, each adapted to carry a font or set of matrices, corresponding fonts of matrices differing in form as between the different fonts, a primary distributer acting on all 30 the matrices to separate them according to letter regardless of font, and a secondary power-driven mechanism for distributing the assorted matrices according to font.

15. In a linotype-machine, a series of 35 magazines, a like number of matrix-fonts, those of each font having a distinguishing form, a preliminary distributing mechanism for separating all the matrices into groups according to letter regardless of font, and a 40 secondary distributing mechanism embrac-ing power-driven switches controlled in their action by the matrices presented thereto.

16. In a linotype-machine, a mechanism for distributing variant fonts of matrices, a 45 preliminary distributer to separate the matrices according to letter, an auxiliary magazine to receive the matrices therefrom, means for releasing the matrices at intervals from the auxiliary magazine, switches controlling 50 the course of the matrices from the auxiliary magazine to the respective magazines, and switch-governing devices controlled by the individual matrices, whereby matrices of the same letter but different fonts are enabled to 55 control the power-driven devices which determine their course to the respective maga-

17. In a mechanism for distributing two or more fonts of matrices, a distributer which 60 delivers all matrices of the same letter, regardless of font, at the same point, a channeled auxiliary magazine arranged to receive all matrices of the same letter in the same channel, means for releasing matrices one at a time

from the magazine-channels, channels each 65 leading from the auxiliary magazine to two or more main magazines, and means controlled by the individual matrices to direct them to the appropriate magazines.

18. In a linotype-machine, the combina- 70 tion of two similar channeled main magazines, an auxiliary channeled magazine, channels each leading from the channels of the auxiliary magazine to corresponding channels of both main magazine escapement de- 75 vices releasing matrices one at a time from the channels of the auxiliary magazine, and means controlled by the individual matrices to control their course to one main magazine or the other.

19. In a linotype-machine, the combination of two similar channeled magazines, two fonts of matrices differing in form, matrixguides, each communicating with corresponding channels of both magazines, a switch in 85 each channel to direct the matrices to one magazine or the other, power-driven devices to actuate said switches, and means controlled by the matrices to determine the ac-

tion of the switches.

20. In combination, magazines B and B', two fonts of matrices dissimilar in form, a distributer S for separating the matrices according to letter, an auxiliary magazine T to receive the matrices from said distributer, 95 escapements to retain the matrices in the auxiliary distributer and release one of each letter at a time, and means controlled by the individual matrices to determine their course to the respective magazines.

21. In combination, the distributer S, auxiliary magazine T, escapements t, t', feelers U. and switches controlled by the feelers to determine the course of the individual mat-

rices.

22. In combination with magazines B, B', inlet-switches  $v^i$ , their actuating-slides  $v^i$ , the slide W adapted to actuate slides  $v^i$  individually, and means controlled by the matrices to govern the movement of slide W.

23. The switches  $v^3$ , slide W, and intermediate connections, in combination with springs w', depressor-arm  $w^3$ , rocking bar  $w^6$ , stepped bar  $u^{ar{i}}$ , feeler U and means for delivering the matrices successively to the feeler.

24. A series of magazines each adapted to carry a font or set of matrices, corresponding fonts of matrices in which all the matrices containing the same character are provided with like distributing features regardless of 120 font, and matrices of different fonts provided with font-distinguishing notches, in combination with a primary distributing mechanism delivering all matrices of the same character to one point regardless of font, and a second-125 ary distributer for separating the groups from the first distributer according to font, for delivery to the respective magazines.

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25. In a linotype-machine, the combination of a series of magazines and corresponding sets of matrices, means for distributing the composed lines of matrices according to letter and accumulating matrices of the same letter regardless of font, and means for distributing the matrices in said groups according to font.

In testimony whereof I hereunto set my hand, this 23d day of April, 1906, in the presence of two attesting witnesses.

FRANK JOHANNESEN.

Witnesses:

HERMAN GERECKE, THOS. E. CUNNINGHAM.