CABLE DRIVEN JOINT ACTUATOR AND METHOD

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Abstract

A cable driven actuator and actuator method involve a movable link that is movable about a path by a cable connected to the link, and a movable support member having a cable routing element. The support member is movable in a manner to change a moment arm of the cable acting on the link to control torque applied to the joint.
CABLE DRIVEN JOINT ACTUATOR AND METHOD

[0001] This application claims benefits and priority of provisional application Ser. No. 60/809,698 filed May 31, 2006, the disclosure of which is incorporated herein by reference.

CONTRACTUAL ORIGIN OF THE INVENTION

[0002] This invention was supported by funding from the Federal Government through the National Institute of Health Science under Grant/Contract No. 5T32 HD 07418. The Government may have certain rights in the invention.

FIELD OF THE INVENTION

[0003] The present invention relates to a cable driven actuator and method incorporating moment arm adjustment features.

BACKGROUND OF THE INVENTION

[0004] In rehabilitation robotic, orthotic, or prosthetic applications, devices have been used to apply forces including torques to various points on the human body in order to manipulate those points. When such devices apply forces or torques under programmable computer control, it is said that the human body is subject to robotic manipulation.

[0005] Current robotic manipulation can be used to provide benefits to clinicians and patients that include, but are not limited to, assessment, motor control studies, and therapy of both healthy people and people with neuromuscular difficulties. However, the robotic machines developed to date have been limited for use in a laboratory setting.

[0006] A robotic machine capable of training or rehabilitating its human user at home or otherwise outside of a laboratory has the potential to be used more often and thus be more effective. Such a robotic machine should be lightweight, inexpensive, and portable, which current rehabilitation robotic machines cannot offer.


[0008] The MIT Manus device uses a five-bar linkage and two torque motors to produce a planar haptic interface (Hogan et al. “MIT-MANUS: a workstation for manual therapy and training”, IEEE International Conference on Robot and Human Communication”, pp. 161-165, Tokyo, Japan 1992). As a linkage, where the individual bars are of fixed length, motion pathways are prescribed by the motions of the joints and by design and size of the linkage.


[0011] A robotic machine that embodies two elastic bands connected to a passive (non-driven) circular disk and that relies on torque unbalance to cause the passive disk to jump between positions is described by Zeeman in “Catastrophe Theory: Selected Papers”, Addison-Wesley 1972-1977.

SUMMARY OF THE INVENTION

[0012] The present invention provides a cable driven actuator mechanism that includes moment arm adjustment features to manipulate the position of the moment arm relative to a movable link.

[0013] In an illustrative embodiment of the present invention, a cable driven joint actuator includes a movable link that can be operatively coupled to a joint to be actuated and that is movable about a path by a cable connected to the link. A cable routing element is provided on a movable support member that is rotated and/or translated in a manner to change the moment arm of the cable acting on the link to control torque applied to the joint. The rotatable pulley support member is rotatable by a first motor. A device is provided to maintain a substantially constant tension on the cable. The device can comprise a cable spool and a second motor to rotate the spool. The pulley on the link and the cable positioning pulley on the movable pulley-support member can be configured as a block and tackle to amplify torque applied to the joint.

[0014] In a particular illustrative embodiment of the present invention, the cable driven joint actuator includes a pivotal link that is adapted to be operatively coupled to a joint to be actuated and that is pivoted about a pivot axis by a length of cable engaging a pulley on the link remote from the pivot axis and having an end coupled to the link. One or more cable positioning pulleys is/are provided on a rotatable pulley-support member that is rotated about an axis that is coaxial with the pivot axis to cause the cable positioning pulley to reposition the cable in a manner to change the moment arm of the cable acting on the link to control torque applied to the joint. The rotatable pulley support member is rotatable by a first motor. A device is provided to maintain a substantially constant tension on the cable. The device can comprise a cable spool and a second motor to rotate the spool. The pulley on the link and the cable positioning pulley on the movable pulley-support member can be configured as a block and tackle to amplify torque applied to the joint.

[0015] The present invention is useful as a robotic training or rehabilitating machine, prosthetic machine, or orthotic machine for human patient use at home or otherwise outside of a laboratory as a result of its being lightweight, inexpensive, and portable.

[0016] The present invention envisions a cable driven actuator for a human limb comprising a cable connected to
a human limb that comprises a pivotal link to be actuated and that is pivoted about an axis by the cable, the cable being connected to the human limb remote from the axis. A movable support includes a cable routing element wherein the support member is movable in a manner to change a moment arm of the cable acting on the human limb to control torque applied to the joint.

[0017] The present invention envisions a cable driven actuator for a garage door or other mechanical link wherein the position of the pivotal arm relative to a mechanical link is manipulated. These other features and advantages of the present invention will be set forth in the following detailed description taken with the following drawings.

DESCRIPTION OF THE DRAWINGS

FIG. 1 is perspective view of a cable driven joint actuator in accordance with an illustrative embodiment of the invention.

FIG. 2 is an enlarged perspective view of the rotator and the cable tensioner of the cable driven joint actuator of FIG. 1.

FIG. 3 is a simplified top view meant to show the variables involved in calculating torque exerted in the joint.

FIG. 4 is a schematic view of a human user grasping the handle for use in training or rehabilitation where the actuator applies a torque about the elbow joint.

FIGS. 5A is a schematic view of a human user having a cable driven actuator to apply torque about the knee joint to move the user's leg. FIG. 5B is an enlarged view of the region boxed-in by dashed lines in FIG. 5A.

FIG. 6 is a view of the opposite side of the knee orthosis.

FIG. 7A and 7B are schematic views of a garage door opener mechanism in “door going up” position, where the moment arm in the cable is set to lift the door up.

DESCRIPTION OF THE INVENTION

In one illustrative embodiment, the present invention provides a cable driven joint actuator mechanism that includes moment arm adjustment features to control torque applied to a joint. The joint to be actuated can include, but is not limited to, a human user’s joint such as an elbow joint, a mechanical joint of a mechanical device, or any other joint.

In a particular embodiment of the present invention offered for purposes of illustration and not limitation with respect to FIGS. 1 and 2, the cable driven joint actuator includes a pivotal link 4 that is adapted to be operatively coupled to a joint to be actuated and that is pivoted about a pivot axis 5 by a discrete length of substantially inelastic cable 12 engaging one or more pulleys 6 disposed on the link 4 remote from the pivot axis 5 and having a cable end coupled to the link as explained below. One or more cable positioning pulleys 8 is/are provided on a rotatable pulley-support member 7 that is rotated about a center axis that is coaxial with the pivot axis 5 to cause the one or more cable positioning pulleys 8 to position the cable in a manner to change the moment arm of the cable acting on the link to control torque applied to the joint. Moment arm is defined using geometry from FIG. 3. The angle of the cable positioning pulleys 8 relative to a datum, Φ, and the angle of the pulleys of link 4 relative to the same datum, θ, are combined with the radius of the link 4 and the cable positioning pulleys 8, Rl, and Rs, respectively. The equation for the moment arm, R, is shown below:

\[ R = \frac{R_l R_s}{\sqrt{R_l^2 + R_s^2 - 2 R_l R_s \cos(\theta - \Phi)}} \]

The rotatable pulley support member 7 is rotatable by a first motor M1. A cable tensioner device 10 is provided to maintain a substantially constant tension on the cable 12. The tensioner device 10 can comprise a cable spool 11 and a second motor M2 to rotate the spool 11. In FIGS. 1, 2, two cable pulleys 6 are shown disposed on the link 4 and two cable pulleys 8 are shown disposed on the pulley-support member 7 configured to form a block and tackle to amplify torque applied to the joint. The various components of the actuator are disposed on a base plate B having a base plate frame E. The end 4a of the link 4 is rotatably mounted between the frame plates E1, E2 of the frame E.

A particular illustrative working embodiment of the invention is now described in more detail with respect to FIGS. 1 and 2. The link 4 rotates about the pivot axis 5 defined by a link pivot shaft 4s sandwiched between two ½ inch inner diameter ABEC 1 bearings from McMaster-Carr Supply Company and mounted between the frame plates E1, E2. The angular position (θ) of the link 4 is measured by a 10 kΩ conductive plastic potentiometer 14 from Spectrum Sensors and Controls, Inc. with a resolution of 0.03° (0.0005 radians). The potentiometer is rotated by the rotatable link shaft 4s that rotates about axis 5.

An adjustable handle 3 is provided and can slide across a track on the link 4 to fit a variety of user arm lengths. Two link pulleys 6 are shown located at the remote end of the link 4 so as to form the distal portion of the cable block and tackle. The pulleys 6 comprise ½ inch outer diameter pulleys from McMaster-Carr Supply Company and are mounted atop another one on the link by a ½ inch diameter steel shaft. All machined components (except for steel shafts) are made of 6061 aluminum alloy.

The pulley-support member 7 comprises a six inch pitch diameter, steel sprocket (Stock Drive Products, Sterling Instrument, 0.25 inch pitch) rotating about its center axis that is coaxial with pivot axis 5 and a roller chain 13 (0.25 inch pitch). The sprocket is rigidly connected to a support hub 7a to prevent wobbling of the sprocket. The member 7 and hub 7a are rotatably mounted on two 0.5 inch inner diameter ABEC 1 bearings from McMaster-Carr Supply Company on a steel shaft 7s fixed to ground (i.e. base plate B). The shaft 4s and the shaft 7s have the same center of rotation. The pulleys 8 (both ½ inch outer diameter) are positioned by a spacer SP to be roughly at the same height as the link 4 for efficient cable-wraping. Each pulley 8 uses a ¼ inch inner diameter ABEC 1 bearing from McMaster-Carr Supply Company. The pulleys 8 are fastened in a fixed position on the member 7 (1.9375 inches from the sprocket center) on fixed shaft 7s. The angular position (θ) of the pulleys 6 is measured by the drive motor M1 with an
The rotator \( r \) is driven by a roller chain 13 and sprocket 15 from Stock Drive Products, Sterling Instrument having a 0.25 inch pitch, 0.6 inch pitch diameter coupled to a drive motor M1, which comprises a Yaskawa AC servomotor (SGM-02B312) with 0.637 Nm continuous torque. The sprocket drive motor M1 is provided with an encoder with 8192 counts/revolution that is used as feedback to measure pulley angle \( \Phi \). Through the transmission ratio of 10, the resulting resolution of the position is \( 0.016^\circ \) (0.0003 radian). The transmission ratio of 10 results from the ratio of the drive motor coupler (not shown) of 0.6 inch diameter to the sprocket (6 inch diameter). Consistent with cable design principles, the angle of incidence of the cable (the fleet angle) does not exceed 2\(^\circ\), the cable does not reverse wrapping, and the pulleys are above the minimum diameter as described by Oberg et al., Machinery’s Handbook, 26th Edition: Industrial Press Inc. which is incorporated herein by this reference to this end.

The rotator \( r \) and the link 4 are mechanically coupled by a steel aircraft cable 12 from Sava Industries (\( 3/8 \) inch diameter, 7x19 strands) that wraps around the rotor pulleys 8 and the link pulleys 6 in a block and tackle configuration to amplify the effective tension of the cable by four, resulting in a four-fold increase in torque and cable excursion. The path of wrapping of the cable from the tensioner device 10 passes through the bottom pulley of the cable positioning pulleys 8, then through the bottom pulley of the link pulleys 6, back to the top pulley of the cable positioning pulleys 8, and then back to the top pulley of the link pulleys 6 until it is anchored back at the shaft 3\( _s \) of the cable positioning pulley 8 by anchor 12\( _b \). To account for the increased excursion, cable tensioner device 10 is provided on the base plate B and comprises a spool 11 driven by a tensioner motor M2, which is also a Yaskawa AC servomotor (SGM-02B312) for multiple cable wraps. The cable 12 wraps around the spool 11 which couples to the tensioner motor M2 with a resolution of 0.16 N, which includes the transmission ratio. Since the cable 12 enters the spool at a large fleet angle but a small fleet angle is desired for better wrapping, a device that decreases the fleet angle at any wrapping level is necessary. This embodiment uses a follower 17 with the same pitch and thread diameter that guides the cable into the spool 11. Since the follower needs to rise and fall with the level of cable on the spool yet maintain consistent orientation, a post 19 is provided with one end fixed to the follower and the other end translatable vertically in the base plate B. The follower 17 is similar to a follower employed on a fishing reel. Proximate one end, the cable 12 runs against the follower 17 and wraps up to the spool 11 as it rotates. Exiting from the follower, the cable needs to match up to the height of the rotator’s pulleys 8. As a result, the cable 12 travels through a cable guidance system that comprises of four pulleys 9 provided to both raise the cable to the proper constant height when approaching the rotator pulleys 8 and also to measure cable tension. The pulleys 9 comprise \( 3/8 \) inch diameter pulleys from McMaster-Carr Supply Company disposed on fixed support block 10\( _b \). There are provided two strain gauges (strain gauge SG 1 being shown on block 10\( _b \) and the other strain gauge being located therebelow on the following block surface 10\( _b \)) that are disposed on the pulley support block 10\( _b \) in a manner to detect cable tension and provide an optional feedback loop with the tensioner motor M2. The strain gauges can comprise 350 \( \Omega \) resistance strain gauges SG from Omega Engineering, Inc. Cables for use in practice of the invention can include, but are not limited to, steel aircraft cable or other substantially inelastic cables. Elastic cables can be used as well such as one or more bungee cords within the scope of the invention. As used herein, the term cable or cables is intended to include a cable, cord, strand, rope, belt, or other substantially inelastic or flexible, elastic elements.

In lieu of the cable being connected to the tensioner device 10 as described above, the cable can be connected to a source of energy storage such as including, but not limited to, a spring. FIG. 7A, 7B, or even an energy dissipation element, such as a damper and bungee cord.

From the above description, it is evident that the drive motor M1 controls the rotational path of the cable positioning pulleys 8 such that the rotator \( r \) is driven remotely, and the other tensioner motor M2 controls the tension in the cable 12. Moreover, the rotator (disk 7 with pulleys 8) and the link 4 rotate independently from another, coupled only by the cable 12.

An advantage of the cable driven joint actuator described above is its simple control strategy. Using a real time operating system, the data comprised of the angular positions of the link 4 and of the rotator \( r \) (disk 7 with pulleys 8) are sampled at 2 kHz. The drive motor M1 which controls the rotator \( r \) is operated in a torque mode, using encoder feedback and controls position. The tensioner motor M2 is operated in open loop torque mode when the strain gauges SG 1, etc. are not used, where a voltage command determines the desired tension in the cable. A general-purpose, procedural, imperative computer programming language, such as C++, and that interrupts in a semaphore structure to control the actuator motors M1 and M2 of FIGS. 1, 2, and 3.

The desired torque to be applied to a joint is created by setting the position of rotator \( r \) to create the proper relative angle between itself and the link 4. For example, the torque per unit tension is the derivative of the excursion according to the position of the link 4 pursuant to: The torque on the arm is the product of the moment arm and the effective tension, which through the block and tackle, is four times the tension:

\[ \tau = 4RT \]

where \( \tau \) is torque, \( T \) is tension in the cable, \( R \) is the moment arm defined above. Endpoint stiffness can be manipulated in the same manner. It is noted that changing the rotator position is equivalent to changing the equilibrium position of the actuator. The link position (determined from the potentiometer) and the rotator position (determined from the motor encoder) are the only feedback components necessary for control of the actuator, since the tension of the cable 12 is held constant in this particular working embodiment. Hard mechanical stops (not shown) are provided to prevent the link 4 from surpassing the user’s range of motion. A chain guard (also not shown) can be provided to cover the exposed portion of the roller chain 13 to prevent any interference.

The cable driven joint actuator described above can be used in an illustrative embodiment as a robotic training or rehabilitating machine, FIG. 4, for a human user who grasps...
the handle 3 on the link 4 so that torque is applied by the actuator about the elbow joint of the user, centered at the pivot point 5. The Table below shows illustrative design parameters for such use. In the Table, the user’s forearm length refers to an actual user’s forearm, on which the length of the link 4 is sized and adjusted, if necessary.

<table>
<thead>
<tr>
<th>Range of Motion from full extension (rad)</th>
<th>User Forearm Length (m)</th>
<th>Torque (N m)</th>
<th>Speed (rad/s)</th>
</tr>
</thead>
<tbody>
<tr>
<td>Minimum</td>
<td>0</td>
<td>0</td>
<td>0</td>
</tr>
<tr>
<td>Maximum</td>
<td>3π/4</td>
<td>0.4</td>
<td>10</td>
</tr>
</tbody>
</table>

The above range of torques is based on a 25 N endpoint force, and the maximum speed is based on an 8 Hz movement. The training or rehabilitation machine can be used in various modes of operation; for example, in a Guidance mode where the actuator torque pushes the user’s arm/hand about the elbow joint toward the desired trajectory of movement using a linear force field of 8 N/m/radian; in an Error Augmentation mode where the actuator torque pushes the user’s arm/hand about the elbow joint away from the desired trajectory of movement using a linear force field of 8 N/m/radian; and in a Control mode where there is no haptic feedback (actuator motor M1 not energized). In summary, the device can be used to control either position or exert any accurate torque on its user as long as the bandwidth and maximum torque are within specifications.

In lieu of using the rotator 7 described above to manipulate the moment arm, the invention envisions using a slide or compound slide (not shown) having one or more cable positioning pulleys disposed thereof to engage and position the cable. The slide or compound slide can be moved linearly by a motor of any type in a direction to manipulate the moment arm. In fact, the invention envisions manipulating the moment arm in any given path, whether it be linear, rotational, or a combination of the two.

Figs. 5A, 5B and 6 are schematic views of a human user having a cable driven actuator to apply torque about the knee joint in a manner to move the user’s leg pursuant to another illustrative embodiment of the invention. The cable driven actuator is attached by straps ST to the leg of the user. Fig. 6 provides a view of the device from the opposite side. Fig. 6 shows a rotator 107 having cable wrapping surface 107w and having a fixed shaft 108a that is connected to a proximal bungee cord anchor 110 which fixes the ends of two bungee cords 112 and that allows the anchor 110 to rotate about the shaft 108a. In this embodiment, the cable routing element is the proximal bungee cord anchor 110. The other ends of the bungee cords are fixed in a distal bungee cord anchor 111 that connects to a fixed shaft 114 distally located on a rigid leg support member 115 in a manner that allows the anchor 111 to rotate about the shaft. The rotator 107 is centered at the knee, and moves in a rotational manner about its rotator shaft, thus moving the proximal bungee cord anchor 110 in a rotational manner.

The position of the rotator 107 is controlled by cable 119 that wraps around the rotator surface 107w and then passes through sheath 119s to a motor M11 on a belt B donned by the user. One end of each cable sheath 119s is anchored to an anchor plate 122 of a rigid thigh support member 124 and referred to as a Bowden sheath anchor. The other end of each sheath 119s is rigidly connected to the motor M11 which wraps the other end of the cable. The members 115, 124 relatively rotate about the rotator shaft during leg movement. The user’s belt B also can include a controller C and power source S, such as a battery pack.

The rotational path of the proximal bungee anchor 110 varies both the length of the bungee cord and the moment arm, altering the torque exerted on the knee. There are two angular position sensors (goniometers) 125 that detect the position of both the rotator 107 and the leg relative to the thigh. Since the torque varies based on rotator position relative to knee flexion angle, the position of the rotator can be varied relative to the leg, and thus a controlled torque can be provided at the knee. The torque could be used for any number of embodiments, including assistive and resistive strategies.

In another illustrative embodiment of the present invention, a cable driven actuator mechanism is provided that includes moment arm adjustment features to manipulate the position of the moment arm relative to a movable link. For purposes of illustration and not limitation, Figs. 7A and 7B show a cable driven joint actuator according to this embodiment for use as a garage door opener device. In this embodiment, an inelastic cable 212 attached on one end to an extension spring S1 fixed to ground, passes through a fixed pulley 214 and then through another pulley 215 attached to a linearly movable bearing 220 for linear movement therewith. The pulley 215 comprises a cable routing element. The linearly movable bearing 220 provides a movable support member for the cable positioning pulley 215. The bearing 220 is moved in linear manner by lead screw 222 driven by motor M11. The cable 212 then attaches to the bottom of a conventional multi-hinged garage door D. The garage door has wheels W that rotate around each hinge and travel along a fixed track T, which provides a path for movement of the garage door. The garage door itself or the door sections is considered a movable link.

The device works by manipulating the moment arm of the cable 212 relative to the position of the door D. To open a closed door, motor M11 moves the linear bearing 220 (with cable positioning pulley 215 thereon) along a horizontal path towards the door, modifying the cable’s line of action it creates with the door and thus the spring tension in the cable in the vertical direction is larger than the weight of the door causing the door to rise. To close an open door, the motor M11 will move the linear bearing 220 (with cable positioning pulley 215 thereon) away from the door until the weight of the door is greater than the vertical direction of the tension in the cable.

While certain embodiments of the invention have been described in detail above, those skilled in the art will appreciate that changes and modifications can be made therein within the scope of the invention as set forth in the appended claims.

What is claimed is:

1. A cable driven actuator, comprising a movable link that is moved about a path in response to a cable connected to the link, and a movable support member having a cable routing
element disposed thereon wherein said support member is movable in a manner to change a moment arm of the cable acting on the link.

2. The actuator of claim 1 wherein said support member is rotatable to cause the cable routing element thereon to change the moment arm.

3. The actuator of claim 2 wherein the support member is rotatable about an axis that is coaxial with a pivot axis of the link.

4. The actuator of claim 1 wherein the support member moves in a translational path to cause the cable routing element thereon to change the moment arm.

5. The actuator of claim 1 wherein the cable is substantially inelastic or elastic.

6. A cable driven joint actuator comprising a pivotal link that is adapted to be operatively coupled to a joint to be actuated and that is pivoted about a pivot axis by a length of cable engaging a pulley on the link remote from the pivot axis and having an end coupled to the link, a pulley-support member having a cable supporting pulley and being rotatable by a motor about an axis that is coaxial with said pivot axis to cause the pulley to position the cable in a manner to change the moment arm of the cable acting on the link to control torque applied to the joint, and a device to maintain a substantially constant tension on the cable.

7. The actuator of claim 6 wherein the tensioner device comprises a cable spool and a second motor to rotate the spool.

8. The actuator of claim 6 wherein the tensioner device comprises a spring or bungee cord.

9. The actuator of claim 6 wherein said pulley of said link and said pulley of said pulley support member form a block and tackle to amplify torque applied to the joint.

10. A cable driven actuator for a human limb, comprising a cable connected to a human limb that comprises a pivotal link to be actuated and that is moved about a pivot axis by the cable, the cable being connected to the human limb remote from the pivot axis, and a movable support member having an axis generally centered on the pivot axis and having a cable routing element disposed thereon wherein said support member is movable in a manner to change a moment arm of the cable acting on the human limb to control torque applied to the joint.

11. A robotic training or rehabilitating machine for a human user comprising the actuator of claim 1 having a handle on the link for grasping by the user.

12. A method of actuating a link, comprising providing a movable link that is movable about a path by a cable connected to the link and moving the link by moving a cable routing element in a manner to change a moment arm of the cable acting on the link.

13. The method of claim 12 including moving the cable routing element in a circular path.

14. The method of claim 12 wherein an axis of the circular path is coaxial with a pivot axis of the link.

15. The method of claim 12 including moving the cable routing element in a linear path.

16. The method of claim 12 wherein the movable link is a human limb.

17. The method of claim 12 wherein the movable link is a mechanical link.

18. The method of claim 12 where the cable routing element fixes an elastic cable to a pulley support member.

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