

April 27, 1965

B. J. GROTZ, JR., ETAL

3,180,408

HEAT EXCHANGER APPARATUS

Filed June 23, 1961

4 Sheets-Sheet 1

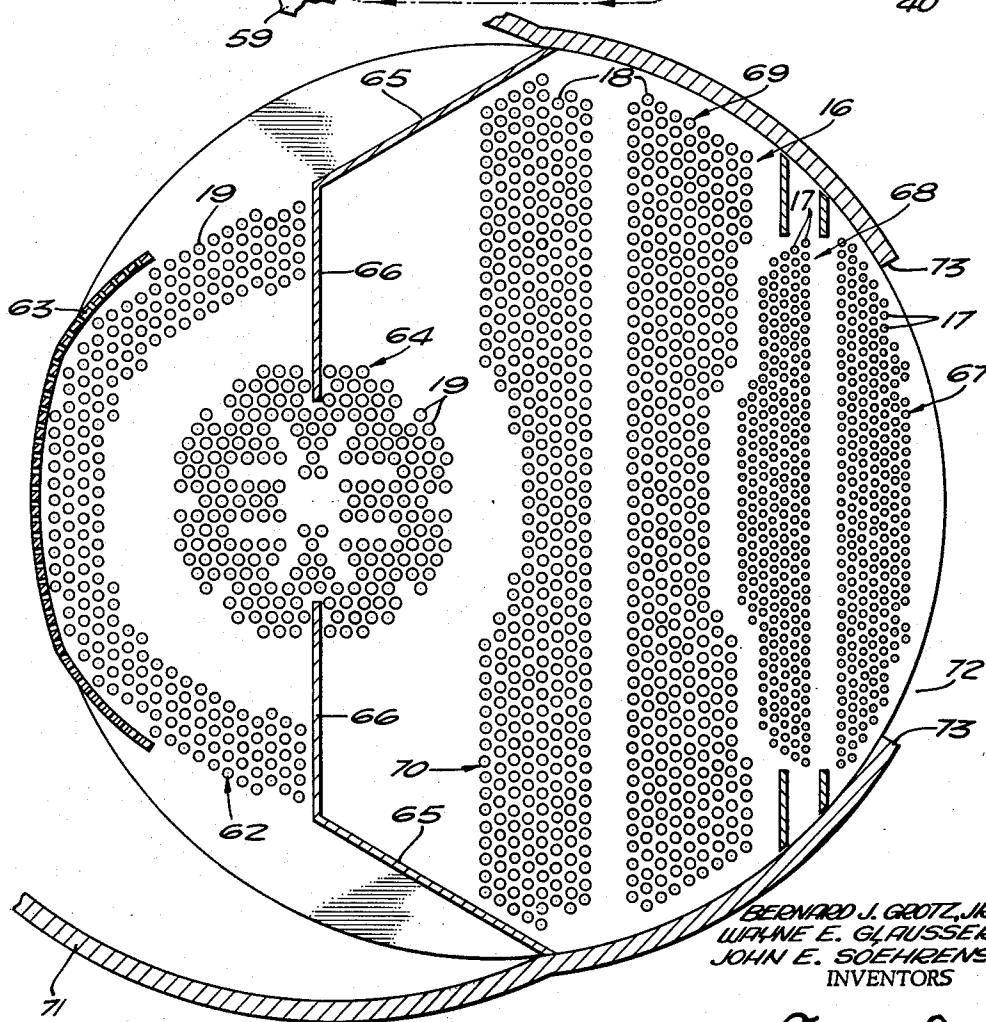
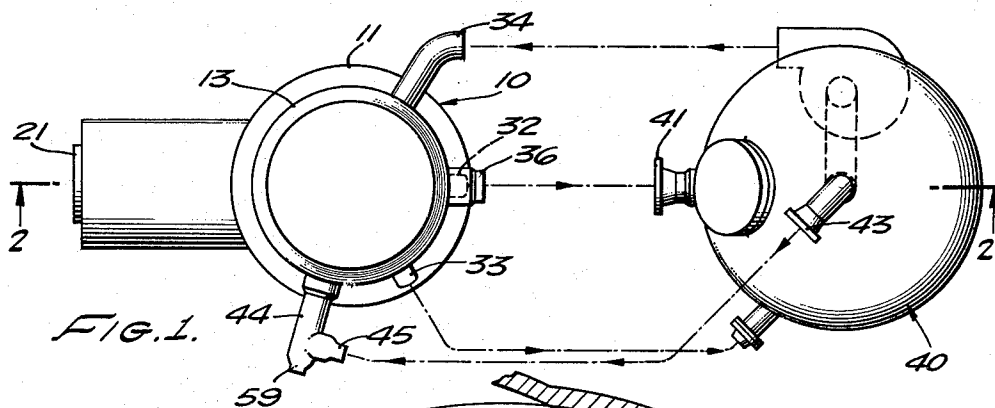


FIG. 3.

**BERNARD J. GROTZ, JR.
WAYNE E. GLAUSSER
JOHN E. SOEHRENS
INVENTORS**

BY

Lyon Lyon

ATTORNEYS

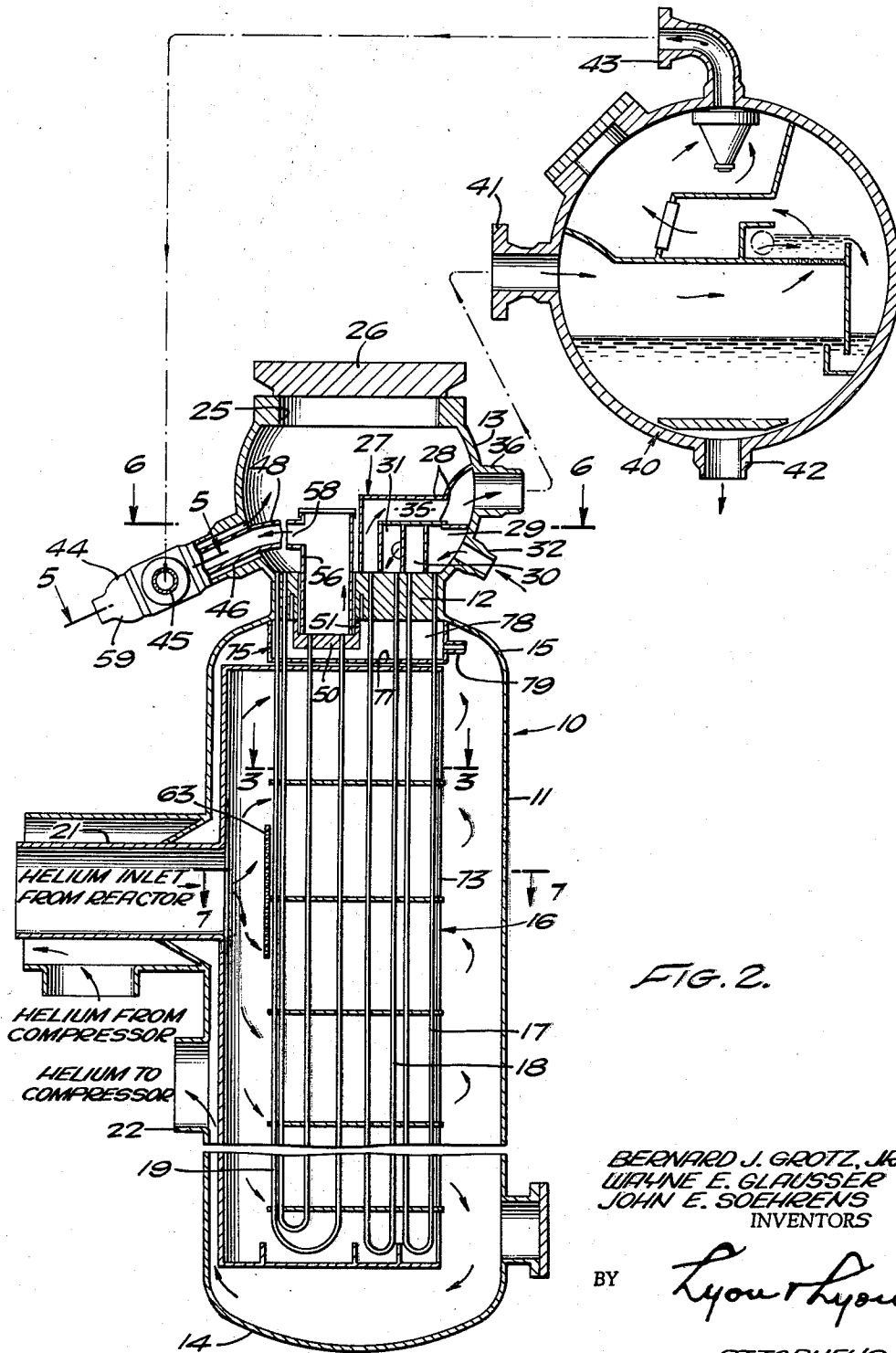
April 27, 1965

B. J. GROTZ, JR., ETAL
HEAT EXCHANGER APPARATUS

3,180,408

Filed June 23, 1961

4 Sheets-Sheet 2



April 27, 1965

B. J. GROTZ, JR., ETAL

3,180,408

HEAT EXCHANGER APPARATUS

Filed June 23, 1961

4 Sheets-Sheet 3

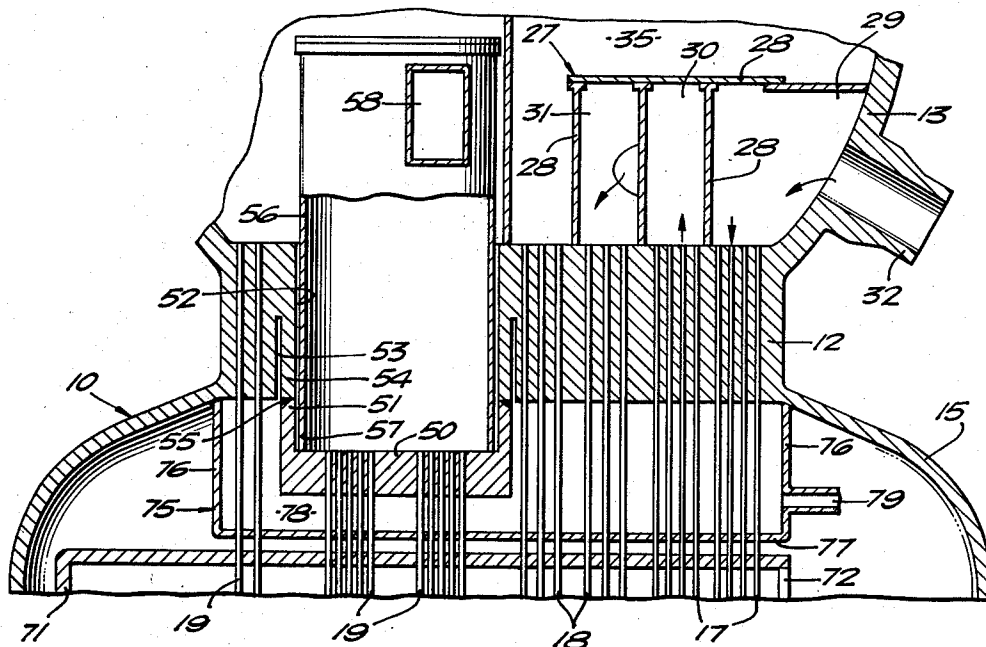


FIG. 4.

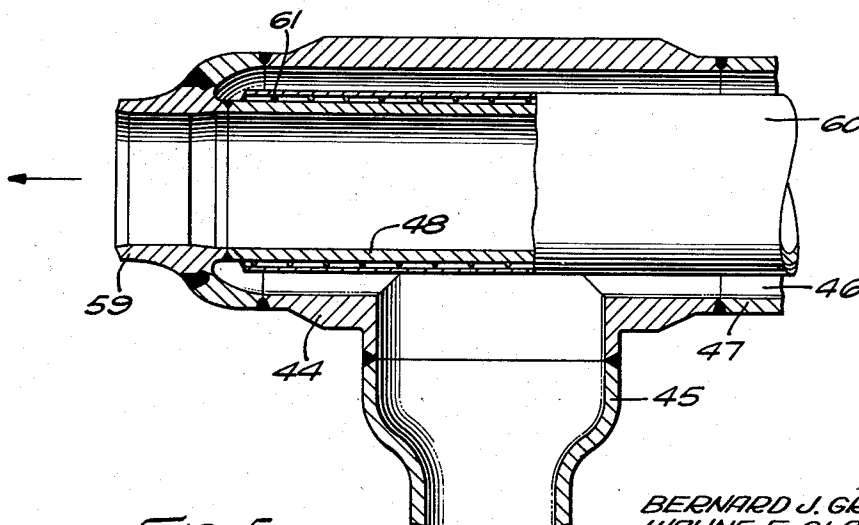


FIG. 5.

BERNARD J. GROTZ, JR.
WAYNE E. GLAUSSER
JOHN E. SOEHRENS
INVENTORS

BY

Lyon & Lyon

ATTORNEYS

April 27, 1965

B. J. GROTZ, JR., ETAL

3,180,408

HEAT EXCHANGER APPARATUS

Filed June 23, 1961

4 Sheets-Sheet 4

FIG. 6.

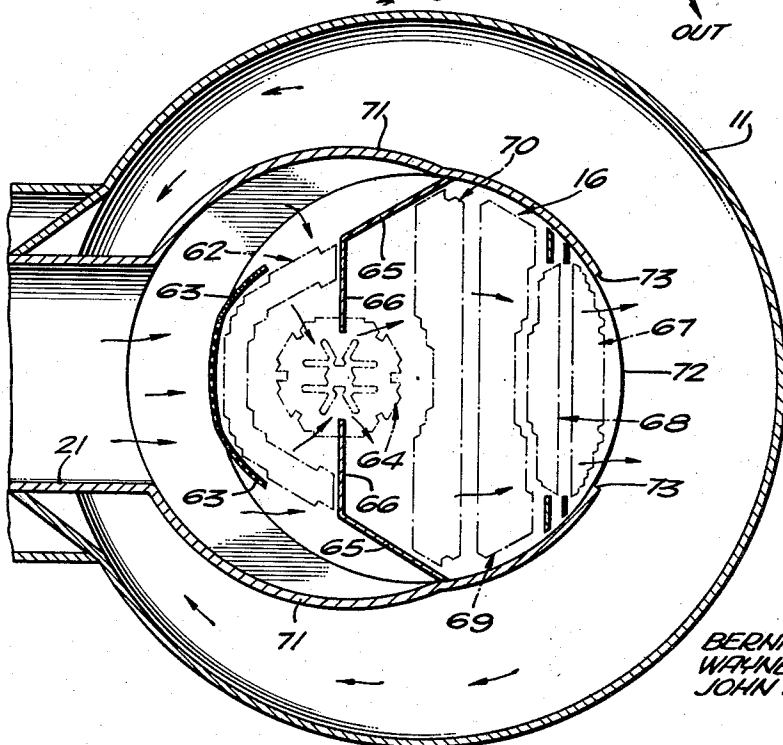
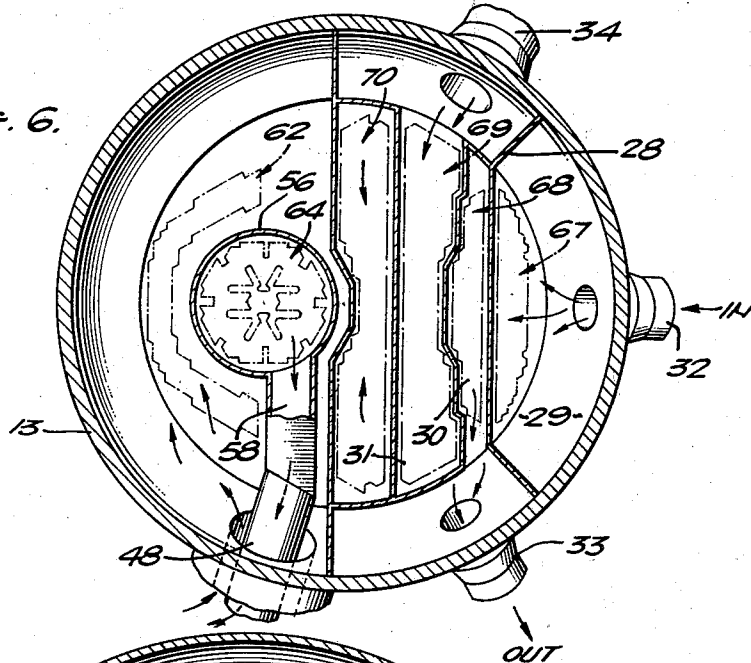


FIG. 7.

BERNARD J. GROTZ, JR.
WAYNE E. GLAUSSER
JOHN E. SOEHRENS
INVENTORS

BY

Lyons Lyon

ATTORNEYS.

1

3,180,408

HEAT EXCHANGER APPARATUS

Bernard J. Grotz, Jr., and Wayne E. Glausser, Arcadia, and John E. Soehrens, Altadena, Calif., assignors to C. F. Braun & Co., Alhambra, Calif., a corporation of California

Filed June 23, 1961, Ser. No. 119,110

8 Claims. (Cl. 165-161)

This invention relates to heat exchange apparatus and is particularly directed to improvements in a steam generator which uses high temperature gas at high flow rates as a source of heat.

It is a principal object of this invention to provide a heat exchanger of the shell and tube type which uses very hot gases in large quantity for the purpose of generating steam. One of the difficulties encountered in providing a steam generator of this type is that the temperature of the superheated steam tubes is so much higher than that of certain other tubes in the tube bundle as to cause difficulty because of differences in thermal expansion. Moreover, it is desirable to make the superheater tubes of a more heat resistant material than that used in the main tubesheet, thereby aggravating the differential expansion problem.

An important object of this invention is to provide a novel form of heat exchanger employing a first tubesheet for connection with a major portion of the tubes in the tube bundle and a second tubesheet fixed in a novel manner to the first tubesheet and being connected to the hot pass portion of the superheater tubes.

Another object is to provide a novel form of tube field pattern for use with said dual tubesheets.

Another object is to provide a means for trapping any leakage of fluids which escape from the tube joints that connect the tubes to the tubesheet and which would otherwise be exposed to the stream of hot gases passing through the shell of the heat exchanger device.

Another object is to provide a novel form of concentric inlet and outlet terminal device for introducing steam into the heat exchanger and for withdrawing superheated steam therefrom.

Other and more detailed objects and advantages will appear hereinafter.

Referring to the drawings:

FIGURE 1 is a top plan view in diagrammatic form showing a preferred embodiment of our invention.

FIGURE 2 is a sectional elevation, taken substantially on the lines 2-2 as shown in FIGURE 1, certain of the connection fittings being turned into the plane of the section for clarity of illustration.

FIGURE 3 is a transverse sectional view, partly broken away, taken substantially on the lines 3-3 as shown in FIGURE 2.

FIGURE 4 is a fragmentary sectional view showing a portion of FIGURE 2 on an enlarged scale.

FIGURE 5 is a sectional detail, partly broken away, taken substantially on the lines 5-5 as shown in FIGURE 2.

FIGURE 6 is a sectional plan view taken substantially on the lines 6-6 as shown in FIGURE 2.

FIGURE 7 is a sectional plan view taken substantially on the lines 7-7 as shown in FIGURE 2.

The heat exchanger or steam generator generally designated 10 includes a shell 11, a tubesheet 12 and a hollow member 13. The tubesheet 12 integrally joins the hollow member 13 to the shell 11. The shell 11 is generally cylindrical in form and is preferably placed in upright vertical position. The shell 11 includes a bottom wall 14 and a tapered portion 15 connected to the tubesheet 12. A tube bundle 16 is mounted in upright position within the shell 11 and this bundle includes a large number of

2

U-bend tubes, including the economizer tubes 17, the evaporator tubes 18 and the superheater tubes 19. Both ends of each of the economizer tubes 17 and the evaporator tubes 18 are fixed in a conventional manner within the tubesheet 12 and communicate with the space within the interior of the hollow member 13. Only one end of each of the superheater tubes 19 is fixed within the tubesheet 12, however, and the other end of each superheater tube 19 is fixed as described below. Hot gas such as helium enters the shell 11 through the inlet pipe 21 and after passing across the tube bundle 16 the cooled helium gas emerges through the outlet pipe 22. After passing through a compressor (not shown) and after being heated, the helium is recirculated and again passes inward through the pipe 21 into the shell 11.

The hollow member 13 may be provided with a flanged opening 25 and this opening may be sealed by means of the closure plate 26 which is normally bolted and then welded in closed position. Flow directing means generally designated 27 are provided within the hollow member 13 and, as shown in the drawings, includes plates 28 forming passages 29, 30, and 31 each communicating with a plurality of tube openings in the tubesheet 12. An inlet fitting 32 on the hollow member 13 allows feed water to enter the passage 29 and to enter the cold pass side of each of the economizer tubes 17. The water emerging from the upper ends of the hot pass sides of the U-bend economizer tubes 17 enters the passage 30 and emerges from the hollow member 13 through outlet 33 (see FIGURE 6). The heated feed water from the outlet nozzle 33 flows to the wash tray in the steam drum. The feed water joins the recirculating water and discharges from the outlet 42. The combined water streams are then pumped to the inlet nozzle 34 on the hollow member 13. This hot water enters the passage 31 which communicates with the inlet pass side of each of the evaporator tubes 18. Wet steam from the outlet pass ends of the evaporator tubes enters the passage 35 and emerges through outlet nozzle 36.

The wet steam from nozzle 36 enters the separate steam drum 40 through inlet nozzle 41. Mechanical apparatus within the steam drum 40 is of conventional type and serves to separate the steam and water and to discharge water through the outlet 42 and saturated steam through the outlet 43. The water from the outlet 42 includes the feed water stream. The steam from the outlet 43 enters the concentric inlet and outlet fitting 44 through lateral terminal 45. The steam enters the interior of hollow member 13 through the passage 46 formed between the outer pipe member 47 and the inner pipe member 48. This steam within the hollow member 13 enters the cold pass side of the superheater tubes 19.

A separate "lowered" tubesheet 50 is provided to receive the other ends of each of the superheater tubes 19. This second tubesheet 50 is formed of heat resistant metal and is generally circular in shape and provided with an upstanding integral rim 51. The tubesheet 12 is provided with an axially extending offset opening 52 and is also provided with a concentric groove 53 defining a depending skirt 54. The upstanding rim 51 of the second tubesheet 50 is welded at 55 to the depending skirt 54 of the first tubesheet 12. The duct 56, formed of heat resistant metal, is fixed to the second tubesheet 50 at its lower end 57. A lateral port 58 in the metal duct 56 connects with the inner pipe 48, so that superheated steam emerging from the tubes 19 passes through the duct 56, lateral port 58 and outward through the inner pipe 48 and through the discharge nozzle 59.

The high temperature superheated steam does not contact the walls of the shell 11 nor the hollow member 13, and does not contact the main tubesheet 12. Moreover, the hot inner pipe 48 does not heat the hollow member

13 to an excessive degree because of the construction of the combined inlet and outlet fitting 44. The inner pipe 48 is preferably enclosed within a tubular sleeve or shield 60 which extends from a location adjacent the discharge nozzle 59 to a position within the interior of the hollow member 13. A continuous wire 61 is wrapped helically on the outer surface of the inner pipe 48 and the shield 60, which is formed of heat resistant metal, is mounted on the wire 61. Thermal insulation is enhanced by the presence of the annular space provided by the wire 61 between the inner pipe 48 and the shield 60.

A novel form of tube field is employed and is best shown in FIGURE 3. The cold pass ends of the superheater tubes 19 are arranged in a half circle or crescent shape field 62 adjacent the perforated impingement baffle 63. The hot pass ends of the superheater tubes 19 terminate in the second or lowered tubesheet 59. These are arranged in a field pattern 64 which is roughly circular. Metal shrouds 65 are provided with portions 66 which project into the circular pattern 64 in a radial direction. The positions of the holes in the lowered tubesheet are such that the pattern 64 allows for substantially uniform heating of the hot pass portions of the superheater tubes 19 as the hot gases within the shell 11 pass from the left side of the baffle portions 66 to the right side thereof, as viewed in FIGURE 3. It will be observed that only one end of each of the superheater tubes 19 is received by the second or lowered tubesheet 59. Both ends of all of the other tubes are fixed to the tubesheet 12. As shown in FIGURE 3, the field pattern 67 relates to the cold pass of the economizer tubes 17 and the field pattern 68 relates to the hot pass of the same tubes. The field pattern 69 relates to the inlet pass of the evaporator tubes 18 and the field pattern 70 relates to the outlet pass of the same tubes. It will be further noted that, while the fluid in the economizer tubes 17 and the evaporator tubes 18 flows in countercurrent relationship with respect to the hot gas in the shell 11, the fluid in the superheater tubes 19 flows in concurrent relationship with such hot gas. In this way, the temperature of the metal of the superheater tubes may be maintained within acceptable limits.

Gas flow through the shell 11 is best shown in FIGURE 7. The gas inlet pipe 21 enters the curved shroud 71 and the gas impinges upon the apertured plate 63, and portions of the gas stream are deflected above and below the plate 63. Gas passes through the semi-circular or crescent shaped tube field 62, and then tube field 64 and between the shroud portions 66. The hot gas then passes through the tube fields 70, 69, 68 and 67, in that order, emerging into the interior of the shell 11 through the gap 72 between the shroud ends 73. It will be observed that the gas does not contact the shell until after passing all of the heat exchange tubes, and hence the shell remains relatively cool. The cooled gas then passes out of the shell 11 through the outlet pipe 22.

Means are provided for preventing any leakage of water or other fluid from the tube joints and passing into the stream of hot gas within the shell 11. For example, in a steam generator employing hot helium gas as a source of heat, it is highly undesirable to allow any water or steam to escape into the helium stream. Accordingly, there is provided a baffle member 75 having side walls 76 and a bottom wall 77. (See FIGURE 4.) This baffle 75 cooperates with the lower side of the tubesheet 12 to form a leakage chamber 73. All of the tubes 17, 18 and 19 pass through openings in the bottom wall 77. Any leakage of water or steam from the openings in the tubesheet 12 or tubesheet 59 is collected within the baffle 75 and is drawn off through a discharge line 79. This line 79 emerges through the wall of the shell 11, at a location not shown.

Having fully described our invention it is to be understood that we do not wish to be limited to the details set

forth herein but our invention is of the full scope of the appended claims.

We claim:

1. In a heat exchanger, the combination of: a shell, a hollow member located at one end of the shell, a first tubesheet integrally connecting the shell and the hollow member, a bundle of U-bend tubes positioned within said shell, certain of said tubes each having both ends fixed to said tubesheet, said tubesheet having an axial opening extending therethrough, a second tubesheet positioned within the shell adjacent the first tubesheet, means including weld means fixing the second tubesheet to the first tubesheet in alignment with said opening, certain other of said tubes each having one end fixed to the first said tubesheet and the other end fixed to said second tubesheet, a duct fixed to the second tubesheet and communicating with said other end of the latter said tubes, said duct extending through said axial opening but spaced from the first tubesheet and projecting into said hollow member, conduit means communicating with said duct and extending exteriorly of said hollow member, and means including an inlet and an outlet on said shell for causing a flow of hot fluid across said bundle of tubes.

2. In a heat exchanger, the combination of: a shell, a hollow member located at one end of the shell, a first circular tubesheet integrally connecting the shell and the hollow member, a bundle of U-bend tubes positioned within said shell, certain of said tubes each having both ends fixed to said tubesheet, said tubesheet having an axial opening extending therethrough and offset from the center of the tubesheet, a second tubesheet positioned within the shell adjacent the first tubesheet, means including weld means fixing the second tubesheet to the first tubesheet in alignment with said opening, certain other of said tubes each having one end fixed to the first said tubesheet and the other end fixed to said second tubesheet, said other end of the latter said tubes terminating short of said first tubesheet and not extending through said axial opening, and means including an inlet and an outlet on said shell for causing a flow of hot fluid across said bundle of tubes.

3. In a heat exchanger, the combination of: a shell, a hollow member located at one end of the shell, a first circular tubesheet integrally connecting the shell and the hollow member, a bundle of U-bend tubes positioned within said shell, certain of said tubes each having both ends fixed to said tubesheet, said tubesheet having an axial opening extending therethrough, a second tubesheet having a peripheral rim and positioned within the shell adjacent the first tubesheet, means including weld means fixing said peripheral rim to the first said tubesheet, certain other of said tubes each having one end fixed to the first said tubesheet and the other end fixed to said second tubesheet, said other end of the latter said tubes terminating short of said first tubesheet and not extending through said axial opening, and means including an inlet and an outlet on said shell for causing a flow of hot fluid across said bundle of tubes.

4. In a heat exchanger, the combination of: a shell, a hollow member located at one end of the shell, a first circular tubesheet integrally connecting the shell and the hollow member, a bundle of U-bend tubes positioned within said shell, certain of said tubes each having both ends fixed to said tubesheet, said tubesheet having an axial opening extending therethrough, a groove in the tubesheet adjacent said bundle and encircling said opening to define a skirt, a second tubesheet having a peripheral rim, weld means joining said skirt to said rim, certain other of said tubes each having one end fixed to the first tubesheet and the other end fixed to said second tubesheet, a duct fixed to the second tubesheet and communicating with said other end of the latter said tubes, said duct extending through said axial opening but spaced from the first tubesheet and projecting into said hollow

member, conduit means communicating with said duct and extending exteriorly of said hollow member, and means including an inlet and an outlet on said shell for causing a flow of hot fluid across said bundle of tubes.

5. In a heat exchanger, the combination of: an upright generally cylindrical shell, a hollow member supported at the upper end of the shell, a first circular tubesheet integrally connecting the shell and the hollow member, and upright bundle of tubes positioned within said shell, said bundle including economizer tubes, evaporator tubes and superheater tubes, each of said tubes comprising U-bends, the economizer tubes and evaporator tubes each having both ends fixed to said tubesheet, said tubesheet having an axial opening offset from the center of said tubesheet, a groove in the tubesheet adjacent said bundle and encircling said opening to define a skirt, a second tubesheet having a peripheral rim, weld means joining said skirt to said rim, said superheater tubes comprising U-bends and each having one end fixed to the first tubesheet and the other end fixed to said second tubesheet, a duct fixed to the second tubesheet and communicating with said other end of the latter said tubes, said duct extending through said axial opening but spaced from the first tubesheet and projecting into said hollow member, conduit means communicating with said duct and extending exteriorly of said hollow member, and means including an inlet and an outlet on said shell for causing a flow of hot fluid across said bundle of tubes, said inlet being located on the side of the shell nearest the position of said second tubesheet.

6. In a heat exchanger, the combination of: an upright shell, a hollow member located at the upper end of the shell, a first horizontal circular tubesheet integrally connecting the shell and the hollow member, an upright bundle of U-bend tubes positioned within said shell, said bundle including economizer tubes, evaporator tubes and superheater tubes, the economizer tubes and evaporator tubes each having both ends fixed to said tubesheet, said tubesheet having an axial opening, a second tubesheet positioned below the first tubesheet, means including weld means fixing the second tubesheet to the first tubesheet below said opening, a duct element within the hollow member and extending through and spaced from said axial opening and fixed to said second tubesheet, said superheater tubes each having one end fixed to the first said tubesheet and communicating with the interior of said hollow member, said superheater tubes each having the other end fixed to said second tubesheet and communicating with said duct element, said other end of the latter said tubes terminating short of said first tubesheet and not extending through said axial opening, and means including an inlet and an outlet on said shell for causing a flow of hot fluid across said bundle of tubes.

7. In a heat exchanger, the combination of: an upright shell, a hollow member located at the upper end of the shell, a first horizontal circular tubesheet integrally connecting the shell and the hollow member, an upright bundle of U-bend tubes positioned within said shell, said bundle including economizer tubes, evaporator tubes and superheater tubes, the economizer tubes and evaporator tubes each having both ends fixed to said tubesheet, said

tubesheet having an axial opening offset from the center of said tubesheet, a groove in the tubesheet adjacent said bundle and encircling said opening to define a skirt, a second circular tubesheet positioned below the first tubesheet and having a peripheral rim, weld means joining said skirt to said rim, a duct element within the hollow member and extending through and spaced from said offset opening and fixed to said second tubesheet, said superheater tubes each having one end fixed to the first said tubesheet and communicating with the interior of said hollow member, said superheater tubes each having the other end fixed to said second tubesheet and communicating with said duct element, said other end of the latter said tubes terminating short of said first tubesheet and not extending through said axial opening, and means including an inlet and an outlet on said shell for causing a flow of hot fluid across said bundle of tubes.

8. In a heat exchanger, the combination of: a shell, a hollow member located at one end of the shell, a first horizontal circular tubesheet integrally connecting the shell and the hollow member, a bundle of U-bend tubes positioned within said shell, said tubesheet having an axial opening offset from the center of said tubesheet, a second circular tubesheet positioned adjacent the first tubesheet remote from said hollow member, means including weld means fixing the second tubesheet to the first tubesheet in alignment with said offset opening, certain of said tubes each having both ends fixed to the first tubesheet, certain other tubes each having one end fixed to the first said tubesheet and having the other end fixed to said second tubesheet, the latter said tubes being positioned at one side of the bundle of tubes, a duct fixed to the second tubesheet and communicating with said other end of the latter said tubes, said duct extending through said axial opening but spaced from the first tubesheet and projecting into said hollow member, conduit means communicating with said duct and extending exteriorly of said hollow member, and means including an inlet and an outlet on said shell for causing a flow of hot fluid across said bundle of tubes, said inlet being adjacent the latter said tubes.

References Cited by the Examiner

UNITED STATES PATENTS

2,133,249	10/38	Kerr	165—176
2,365,878	12/44	Huff	165—163
2,457,022	12/48	Yula	165—106
2,576,309	11/51	Ruemelin	165—141
2,730,337	1/56	Roswell	165—70 X
2,774,575	12/56	Walter	165—163 X
2,853,277	9/58	Boni et al.	165—11
2,880,973	4/59	Langrand	165—83
2,881,012	4/59	Rings	285—30
2,892,641	6/59	Ford	285—30
2,920,874	1/60	Gardner	165—140
3,020,024	2/62	Lawrence	165—82 X

CHARLES SUKALO, *Primary Examiner.*

60 HERBERT L. MARTIN, *Examiner.*