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(54) **A lock mechanism**

Verriegelungsmechanismus

Mécanisme de verrouillage

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Description

[0001] The invention concerns a lock mechanism for a door. The lock mechanism is prepared for receiving a lock cylinder with rotatable engagement means.

[0002] Such lock mechanisms are known from e.g. WO 2004/040086 A1, and are typically housed in a lock case secured to a lock post which is fixedly mounted to a door.

[0003] Another lock mechanism is known from US 3 175 376 A, which discloses a mechanism according to the preamble of claim 1.

[0004] According to some standards it should be possible to rotate the key for 360 degrees when it is inserted in the lock cylinder. Furthermore, for many applications it is important to have a relatively large displacement of the bolt. For example some standards demand a minimum of 20 mm bolt displacement. This displacement must be achieved by one rotation of the key, i.e. 360 degrees. If more than one rotation of the key is needed to achieve the 20 mm displacement, this may not always be performed in practice and this therefore implies a security risk.

[0005] Another issue is that the direction of rotation of the key should be logical to the user. Most users find that when locking a door, the upper part of the key should be moved towards the door post or in other words in the direction in which the bolt extends from the lock mechanism. Furthermore, it is advantageous if the same lock mechanism can be used with different types of lock cylinders. Two examples of common lock cylinders are the "Euro Profile cylinder" and the "Scandinavian Oval cylinder".

[0006] All this has been achieved by a locking mechanism arranged according to claim 1.

[0007] In a preferred embodiment of the invention the engagement can also be rotated in the opposite direction whereby the engagement means will engage with the receiving means at the second point on said curved path and disengage from the receiving means at the first point on said curved path.

[0008] In a preferred embodiment of the invention the engagement means could be a protrusion. This will be the case for the Euro Profile cylinder. When the engagement means is a protrusion, the receiving means can be a recess in the toothed element.

[0009] In a preferred embodiment, the toothed element of the lock mechanism can be arranged to be slidable along said curved path. This offers a simple and reliable construction.

[0010] In a further embodiment the part of said curved path along which the engagement means could be in engagement with the receiving means, could have an average radius which is larger than the radius of rotation of said engagement means, and which has a centre different from the centre of rotation of said engagement means. This is a simple and reliable arrangement for ensuring an accurate engagement and disengagement between the engagement means and the receiving means.

[0011] In a further embodiment the toothed wheel can be connected with a tap which extends perpendicular to the plane of the toothed wheel into a slit connected to said bolt, such that when the toothed wheel is rotated, the rotational movement of the wheel is transferred to a rectilinear or substantially rectilinear displacement of said bolt. This is a simple and reliable arrangement for transferring the rotational movement of the toothed wheel to a displacement of the bolt. The toothed wheel including the tap is furthermore a simple way of transferring the movement of the toothed element to the displacement of the bolt.

[0012] In a further embodiment of the invention the slit could extend in its longitudinal direction transversely to the direction of displacement of the bolt. Thereby the component of the movement of the tap parallel with the direction of displacement of the bolt is transferred directly to the bolt. At the same time, due to the slit, the component of the movement transverse to the displacement of the bolt will not influence the displacement of the bolt.

[0013] In a further embodiment the lock mechanism is provided with means for preventing the bolt from being pressed into the lock case when the bolt is in a fully extended position. Thereby, the security of the lock mechanism is further improved.

[0014] In a further embodiment, the means for preventing the bolt from being pressed into the lock case is the tap connected with said toothed wheel and extending into said slit. Preferably the tap has an end position when the bolt is fully extended from the lock mechanism, said end position is placed on or above a line being parallel or substantially parallel with the direction of displacement of said bolt and passing through the centre of rotation of said toothed wheel, thus obtaining a simple and reliable solution.

[0015] In a further embodiment the position of said toothed element is biased such that the toothed element has two stable positions where the engagement means is disengaged from said receiving means. This will prevent accidental movement of the toothed element when the engagement means is disengaged from the receiving means. Such biasing could be by means of a spring load.

[0016] Embodiments of the invention are now described in further detail with reference to the figures. These embodiments are not limiting for the scope of protection defined in the claims.

[0017] Figure 1 shows a perspective view which illustrates the main parts of the lock mechanism arranged in a lock case.

[0018] Figure 2 illustrates an exploded view of the main parts for the lock mechanism.

[0019] Figure 3 shows a side view which illustrates the lock mechanism with the bolt fully extended.

[0020] Figure 4 shows a side view which illustrates the lock mechanism with the bolt in an intermediate position.

[0021] Figure 5 shows a side view which illustrates the lock mechanism with the bolt fully withdrawn.

[0022] Figure 6 is equivalent to figure 3 but for a Scan-

dinavian Oval cylinder.

[0023] Figure 7a - d illustrates specific parts for the lock mechanism in figure 6.

[0024] Figure 8 is equivalent to figure 4 but for a Scandinavian Oval cylinder.

[0025] Figure 9 is equivalent to figure 5 but for Scandinavian Oval cylinder.

[0026] It should be noted that terms of orientation, for example above, clockwise etc. are to be interpreted according to the orientations of the figures. It should be understood by the person skilled in the art that the devices could be installed in other orientations.

[0027] Figures 1 - 5 show a lock for a Euro Profile cylinder. This lock cylinder is drop shaped, and is extending through the lock case 1 from one side to the other. At an intermediate point of this drop shaped lock cylinder engagement means 5 are provided which are arranged such that they can perform a 360 degrees rotation.

[0028] In figures 1 - 3 the lock case 1 with one cover removed is shown. The lock mechanism is arranged within the lock case 1 comprising two covers each covering one of the major surfaces. The lock case 1 is connected to a lock post 2 which is to be secured to e.g. a door. The lock post 2 is provided with holes 8 for e.g. screws to secure the lock post 2 to a door. Within the lock case 1 the lock mechanism is arranged with an opening 6 for a lock cylinder 4. The lock cylinder will be extending out of the lock case 1 perpendicular to the plane of the paper when inserted in the opening 6. A rotational movement of a key inserted in the lock cylinder will cause rotation of engagement means 5, which are arranged such that they can be rotated 360 degrees. The engagement means 5 are in this case a protrusion in the shape of a tooth.

[0029] The opening 6 for the lock cylinder 4 is formed by a first 18 and a second 19 plate.

[0030] The lower part 9 of the lock case 1 illustrated in the figures is typically used for a handle and a latch controlled by this handle. This is, however, not essential for the invention and is therefore not shown.

[0031] The lock mechanism is further provided with a toothed element 11 movable along a curved path 13 and connected with receiving means 12. The receiving means 12 will in this case have the form of a recess, e.g. a notch, fitting together with a protruding engagement means 5. The engagement means 5 and the receiving means 12 are formed to fit together, such that when they are in engagement, rotation of the engagement means 5 will give rise to a displacement of the toothed element 11 along the curved path 13.

[0032] The curved path in which the toothed element 11 is movable is in this case formed as a groove 23 in said first plate 18 and in said second plate 19. The toothed element 11 is slidable in this groove 23 via a protrusion formed on the toothed element 11. The toothed element 11 is further in engagement with a toothed wheel 15 which is rotatably mounted. The toothed wheel 15 has a tap 16 eccentrically connected to the toothed wheel 15. This tap

extends into a vertical slit provided in a base plate 30 of a bolt 3.

[0033] A displacement of the toothed element 11 will cause a rotation of the toothed wheel 15, which by means of the tap 16 and the slit 20 will cause a displacement of the bolt 3.

[0034] The bolt 3 is connected to the base plate 30 into which the slit 20 is arranged. The base plate, being part of the bolt 3, is prepared for being displaced within the lock case 1 following movement of the tap 16 connected to the toothed wheel 15, when the tap 16 is moved in the slit 20. As illustrated in figure 3 the base plate 30 is placed in front of the toothed wheel 15, and the slit 20 in the base plate 30 is preferably arranged such that the tap can be rotated with the toothed wheel for approximately 180 degrees.

[0035] Figures 3-5 show three different positions of the bolt 3 depending on the position of the engagement means 5, and thereby on the position of a key in a lock cylinder.

[0036] Figure 3 shows the lock mechanism with the bolt fully extended from the lock case 1. In this position the toothed element 11 has been moved as far anticlockwise as possible, i.e. into the position opposite the direction of extension of the bolt 3 from the lock case 1. This means that the teeth of the toothed element 11 in engagement with the toothed wheel 15 are the outer teeth at one end of the row of teeth on the toothed element 11. This is the position where the engagement means 5 cannot move the toothed element 11 further in an anticlockwise direction. The engagement means 5 will lose contact or engagement with the receiving means 12 of the toothed element 11 when the engagement means 5 is moved further in an anticlockwise direction according to the figures. The engagement means 5 may, therefore, be rotated further in an anticlockwise direction without being engaged with the receiving means 12. This enables the key to rotate 360 degrees. The rotation stops when the engagement means 5 abuts the toothed element 11 again. This is shown by the tap shown with dotted lines in figure 3.

[0037] With the position of the toothed element 11 shown in figure 3, the position of the toothed wheel 15 and the bolt 3 will also be given. A spring 21 is arranged between one fixed point of rotation on the lock case 1 and one point connected to the toothed wheel 15 such that the spring 21 will exert a force holding the toothed wheel 11 in the position shown in figure 3 where the bolt 3 is fully extended. In this position the toothed element 11 is abutting against an end of the groove 23 preventing also the toothed wheel 15 from further clockwise rotation. The purpose is to prevent any movement of the toothed element 11 when the receiving means is not in engagement with the engagement means 5. It should be mentioned that the friction in the mechanism should be enough, but a spring is useful to provide extra security.

[0038] When the bolt 3 is in this fully extended position as shown in figure 3 the tap 16 connected to the toothed

wheel 15 will be positioned such that any attempt to force the bolt 3 into the lock case 1 will result in a force on the tap 16 directed directly towards the centre of rotation of the toothed wheel 15. Therefore there will be no component of this force acting tangentially on the toothed wheel, and therefore it cannot be rotated by such a force on the bolt. The lock mechanism is therefore self-locking.

[0039] Figure 4 shows the lock mechanism with the bolt in an intermediate position where it is being moved between the fully extended and the fully withdrawn positions. In this position the teeth of the toothed element 11 which are in engagement with the toothed wheel 15, are some of the middle teeth in the row of teeth on the toothed element 11. In this position, the toothed element 11 will follow movement of the engagement means 5 in both clockwise and anticlockwise direction. The spring 21 will exert a force on the toothed wheel seeking to rotate it into one of the two end positions where the bolt 3 is either fully extended or fully withdrawn. The turning point which decides which position the spring load will seek to force the toothed wheel into, is when the tap is halfway between the two stable positions.

[0040] It is seen from figures 3 - 5 that due to the dimensions and placement of the slit 20 it is only possible to move the bolt 3 from fully extended to fully withdrawn by rotating the toothed wheel 15 anticlockwise. In figure 3 the tap 16 is prevented by the slit 20 from being moved upwards. Likewise, it is only possible to move the bolt from fully withdrawn to fully extended by rotating the toothed wheel 15 clockwise. In figure 5 the tap 16 is prevented by the slit 20 from being moved upwards.

[0041] Figure 5 shows the lock mechanism with the bolt fully withdrawn in the lock mechanism. In this position the toothed element 11 has been moved as far clockwise as possible in the groove 23 forming the curved path 13, i.e. into the position in the direction where the bolt 3 is to be extended from the lock case 1. Further clockwise rotation of the engagement means 5 will cause the engagement means 5 to lose contact or engagement with the receiving means 12 of the toothed element 11. The engagement means 5 are therefore free to complete their 360 degrees rotation.

[0042] In this position of the key, the bolt 3 is fully withdrawn. The spring 21 will be holding the toothed wheel 11 in the stable position shown in figure 5. In this position the toothed element 11 is abutting against the other end of the groove forming the curved path 13 preventing also the toothed wheel 15 from further anti clockwise rotation. Furthermore, the tap 16 abuts against the top of the slit 20 also preventing further anticlockwise rotation of the toothed wheel.

[0043] It is seen from figures 3 - 5 that the movement between the toothed element 11 and the toothed wheel ensures that turning the upper part of the key towards the lock post 2 will extend the bolt 3 from the lock case 1, thereby locking the door in which the lock mechanism has been arranged.

[0044] Correspondingly, turning the upper part of the

key away from the lock post 2 will unlock the door by withdrawing the bolt 3 into the lock case 1. This is also the turning direction of the key for locking and unlocking, respectively, which intuitively will be the most logical turning direction for most people.

[0045] In figures 3 - 5 the circle 14 along which the engagement means 5 will rotate is indicated with dotted lines. It is seen that the radius of this circle is clearly smaller than the radius of the curved path 13 of the receiving means 12. Also, the centre of the curved path 13 is located differently than the centre for the rotation of the engagement means 5. This arrangement of different radius and different centre of rotation facilitate that the engagement means 5 are only in engagement with the receiving means 12 for a part of a 360 degrees full circular rotation. This part of engagement is for the Euro Profile cylinder 110 degrees of the rotational movement of the engagement means 5 and for the Scandinavian Oval cylinder this part of the engagement is 125 degrees of the rotational movement of the engagement means 5. This means that 110 or 125 degrees of the rotation of the key is applied for displacement of the bolt 3, and during this part of the rotation there will be proportionality between the rotational position of the key and the position of the bolt 3. In a preferred embodiment of the lock mechanism the total displacement of the bolt will be 20 mm or approximately 20 mm.

[0046] The curved path 13 may have other shapes than part of a circle as illustrated. It may be formed from different circle parts having different radius and different centres. The important point is that when the engagement means 5 are rotated in one direction the engagement means 5 will engage with the receiving means 12 at one point of said curved path 13 and disengage from the receiving means 12 at a different point of said curved path.

[0047] The toothed element 11 is preferably formed as a sliding part being slideable in the curved path 13. Alternatively, the toothed element 11 is pivotable about a point such that it is movable in the curved path 13.

[0048] It can also be seen from figures 3-5 that the lock cylinder is held in place by a screw 31 which is inserted through a hole 32 in the lock post 2. This screw is standard for all Euro-Profile cylinders and the centre axis B of this screw is according to the standard of the Euro-profile cylinder located perpendicular to and offset 19 mm from the centre axis A of the key of the lock cylinder as well as perpendicular to the lock post 2 of the lock mechanism. In order for the lock mechanism to function with a Euro-Profile cylinder, the toothed element and the curved path on which the toothed element slides, are arranged such that the toothed element is located such that it does not conflict with this screw during the normal operation of the lock mechanism.

[0049] One way of doing this is to arrange the toothed element and the curved path such that the toothed element, during normal operation, is entirely located on one side of a plane which is arranged tangent to the screw.

In this way, the toothed element will not come into contact with the screw. Since the screw used is an M5 (5mm diameter) screw, this plane P will be located at an offset of 16.5 mm from the centre axis of the key of the lock cylinder. In the embodiment shown, the plane is arranged parallel to the centre axis of the key of the lock cylinder and is arranged perpendicular to the plane of symmetry of the lock cylinder. Another way of saying this is to say that the toothed element is arranged on one side of the screw which holds the euro-profile cylinder in place in the lock mechanism housing. It can also be seen in the current embodiment, that the toothed element is arranged on the same side of the plane P as the key of the lock cylinder. It can also be mentioned that in the current embodiment, the plane is located between the centre axis A of the key of the lock cylinder and the centre axis B of the screw 31. It could also be said that the plane P is located on the same side of the centre axis of the lock cylinder as the engagement means in the position of the lock cylinder in which the key can be removed from the lock cylinder.

[0050] It should be noted that this is important for the current invention, since it is often desired to use a Euro-Profile cylinder in a lock mechanism. This is especially true in Europe. Many other forms of locking mechanism previously available are not suitable for use with a Euro-profile cylinder. For example US 3,175,376 shows a ring formed toothed element which encircles the lock cylinder. Such a ring formed toothed element will not work with a euro-profile cylinder since the toothed element will conflict with the screw for holding the Euro-Profile cylinder. The ring formed toothed element of US 3,175,376 will only be suitable for use together with a US type lock cylinder.

[0051] In figure 6 - 9 a Scandinavian Oval cylinder is used. This type of lock cylinder does not extend through the lock case 1. This means that two different cylinders can be entered into the opening 6 for cylinders, one from each side of the lock case 1. Inside the opening 6 for cylinders a transferring means 25 (shown in figure 7b) is arranged for transferring the rotational position of both of the two cylinders to the engagement means 5. The transferring means is provided with a cross recess or a cross opening for correspondence with means on the cylinders rotatable with e.g. the key. The transferring means 25 may comprise the engagement means 5 or is in contact with the engagement means 5 when inserted in the opening 6 for cylinders. It should be noted that figure 7b shows a single transfer means 25. In the actual device of figure 6, 8 and 9, there are two transfer means 25, one being a mirror image of the other. The two transfer means are arranged in the lock mechanism such that the two engagement means 5 point towards each other. In this way, the two lock cylinders can operate independently of each other.

[0052] Often the cylinder accessible from outside the door is to be opened by a key, while the cylinder accessible from inside the door is rotatable by a simple knob.

Both cylinders have means for engaging with the transferring means 25.

[0053] When assembling the lock mechanism the transferring means 25 is arranged in the lock case 1, in the area between the first 18 and the second 19 plate. Note that the first and second plate 18 and 19 of the embodiment shown in figures 1-5 and the first and second plate 18 and 19 of the embodiment shown in figures 6-9 are different due to the differences in the locking cylinders. By exchanging the first and second plate of the embodiment of figures 1-5 with the transfer means 25 and the first and second plate of the embodiment shown in figures 6-9, the locking mechanism can easily be changed from a Euro-Profile cylinder lock mechanism to a Scandinavian Oval cylinder lock mechanism.

[0054] It should also be noted that as the person skilled in the art will know, a Scandinavian Oval cylinder as shown in this embodiment is held in place in the lock mechanism by two screws (not shown) having axes which are parallel to the centre axis of the lock cylinder and being arranged in the portion of the lock cylinder above the key (according to the orientation shown in the figures). These screws also impose significant restrictions on the design of the lock mechanism since no moving parts can be in conflict with these screws. It is therefore a significant challenge to provide a lock mechanism which can work with both a Scandinavian Oval cylinder as well as a Euro-Profile cylinder.

[0055] In the embodiments shown in the figures the bolt 3 has been an integral component of the lock mechanism and has been arranged inside the lock case and is extendable from the lock case 1. However, it should be clear to the person skilled in the art that the bolt 3 could also be arranged distant from the lock case 1. The position of the bolt 3, i.e. extended or withdrawn, could then be controlled via a rod controlled by the position of the toothed wheel 15. A bolt 3 placed in the lock case 1 as shown in the figures could also be supplemented by one or two further external bolts controlled through one or two rods.

Claims

1. A lock mechanism for a door or window, said lock mechanism being prepared for receiving a lock cylinder 4 with engagement means 5 which are rotatable 360 degrees, said lock mechanism comprising

- a toothed element 11 being movable along a curved path 13 and having receiving means 12 prepared for engaging with said engagement means 5, said curved path 13 having a shape such that when the engagement means 5 are rotated in one direction the engagement means 5 will engage with the receiving means 12 at a first point on said curved path 13 and disengage from the receiving means 12 at a second point

on said curved path 13, such that the engagement means 5 can rotate 360 degrees,
 - a toothed wheel 15 in engagement with said toothed element 11 such that displacement of said toothed element 11 along said curved path 13 causes a rotation of said toothed wheel 15; and
 - a bolt 3 displaceable by a rotational movement of said toothed wheel 15,

characterized in that said toothed element being arranged such that during normal operation of the lock mechanism, the toothed element is entirely located on one side of a plane which is arranged perpendicular to the plane of symmetry of the lock cylinder and parallel to the centre axis of the key of the lock cylinder and offset from the centre axis of the key of the lock cylinder by 16.5 mm.

2. A lock mechanism according to claim 1, wherein when the engagement means 5 are rotated in the opposite direction, the engagement means 5 will engage with the receiving means 12 at the second point on said curved path 13 and disengage from the receiving means 12 at the first point on said curved path 13.
3. A lock mechanism according to claim 1 or 2, wherein said engagement 5 means is a protrusion, and said receiving means 12 is a recess in the toothed element 11.
4. A lock mechanism according to any one of the previous claims wherein said toothed element 11 is arranged to be slidable along said curved path 13.
5. A lock mechanism according to any one of the previous claims, wherein the part of said curved path 13 along which the engagement means 5 will be in engagement with the receiving means 12, has an average radius which is larger than the radius of rotation of said engagement means 5, and which has a centre different from the centre of rotation of said engagement means 5.
6. A lock mechanism according to any one of the previous claims, wherein said toothed wheel 15 is connected with a tap 16 extending perpendicular to the plane of the toothed wheel 15 into a slit 20 connected to said bolt, such that when the toothed wheel 15 is rotated, this rotational movement is transferred to a rectilinear or substantially rectilinear displacement of said bolt 3.
7. A lock mechanism according to claim 6, wherein said slit 20 is extending in its longitudinal direction transversely to the direction of displacement of said bolt 3.

8. A lock mechanism according to any one of the previous claims, wherein the lock mechanism is provided with means for preventing the bolt 3 from being pressed into the lock case 1 when the bolt 3 is in its extended position.
9. A lock mechanism according to claims 6 - 8, wherein said means for preventing the bolt 3 from being pressed into the lock 1 case is the tap 16 connected with said toothed wheel 15 and extending into said slit 20.
10. A lock mechanism according to claim 9, wherein said tap 16 has an end position when the bolt 3 is fully extended from the lock mechanism, said end position is placed on or above a line being parallel or substantially parallel with the direction of displacement of said bolt 3 and passing through the centre of rotation of said toothed wheel 15.
11. A lock mechanism according to any one of the previous claims, wherein the position of said toothed element 11 is biased in its two end positions, such that the toothed element 11 has two stable positions where the engagement means 5 is disengaged from said receiving means 12.

Patentansprüche

1. Schlossmechanismus für eine Tür oder ein Fenster, der geeignet ist, einen Schließzylinder 4 mit Eingriffsmitteln 5 aufzunehmen, die um 360 Grad drehbar sind, wobei der Schlossmechanismus Folgendes umfasst:
 - ein gezahntes Element 11, das entlang einer gebogenen Bahn 13 beweglich ist und ein Aufnahmemittel 12 aufweist, das geeignet ist, die Eingriffsmittel 5 in Eingriff zu nehmen, wobei die gebogene Bahn 13 eine derartige Form hat, dass die Eingriffsmittel 5, wenn sie in eine Richtung gedreht werden, das Aufnahmemittel 12 an einem ersten Punkt auf der gebogenen Bahn 13 in Eingriff nehmen und aus dem Aufnahmemittel 12 an einem zweiten Punkt auf der gebogenen Bahn 13 ausrücken, so dass sich die Eingriffsmittel 5 um 360 Grad drehen können,
 - ein Zahnrad 15 in Eingriff mit dem gezahnten Element 11, so dass die Verschiebung des gezahnten Elements 11 entlang der gebogenen Bahn 13 eine Drehung des Zahnrads 15 bewirkt, und
 - einen Riegel 3, der durch eine Drehbewegung des Zahnrads 15 verschiebbar ist,

dadurch gekennzeichnet, dass das gezahnte Element so angeordnet ist, dass es sich während des

normalen Betriebs des Schlossmechanismus gänzlich auf einer Seite einer Ebene befindet, die senkrecht zur Symmetrieebene des Schließzylinders und parallel zur Mittelachse des Schlüssels des Schließzylinders und 16,5 mm von der Mittelachse des Schlüssels des Schließzylinders versetzt verläuft.

2. Schlossmechanismus nach Anspruch 1, wobei die Eingriffsmittel 5, wenn sie in die entgegengesetzte Richtung gedreht werden, das Aufnahmemittel 12 an einem zweiten Punkt auf der gebogenen Bahn 13 in Eingriff nehmen und aus dem Aufnahmemittel 12 am ersten Punkt auf der gebogenen Bahn 13 ausrücken.
3. Schlossmechanismus nach Anspruch 1 oder 2, wobei das Eingriffsmittel 5 ein Vorsprung und das Aufnahmemittel 12 eine Ausnehmung im gezahnten Element 11 ist.
4. Schlossmechanismus nach einem der vorhergehenden Ansprüche, wobei das gezahnte Element 11 so angeordnet ist, dass es entlang der gebogenen Bahn 13 gleiten kann.
5. Schlossmechanismus nach einem der vorhergehenden Ansprüche, wobei der Teil der gebogenen Bahn 13, entlang dem das Eingriffsmittel 5 mit dem Aufnahmemittel 12 in Eingriff steht, einen Durchschnittradius hat, der größer als der Drehungsradius des Eingriffsmittels 5 ist, und eine Mitte hat, die sich von der Drehungsmitte des Eingriffsmittels 5 unterscheidet.
6. Schlossmechanismus nach einem der vorhergehenden Ansprüche, wobei das Zahnrad 15 mit einem Zapfen 16 verbunden ist, der sich senkrecht zur Ebene des Zahnrads 15 in einen mit dem Riegel verbundenen Schlitz 20 erstreckt, so dass die Drehbewegung, wenn das Zahnrad 15 gedreht wird, in eine geradlinige oder im Wesentlichen geradlinige Verschiebung des Riegels 3 übertragen wird.
7. Schlossmechanismus nach Anspruch 6, wobei sich der Schlitz 20 in seiner Längsrichtung quer zur Verschiebungsrichtung des Riegels 3 erstreckt.
8. Schlossmechanismus nach einem der vorhergehenden Ansprüche, wobei der Schlossmechanismus mit einem Mittel versehen ist, das verhindert, dass der Riegel 3 in den Schlosskasten 1 gedrückt wird, wenn der Riegel 3 in seiner ausgefahrenen Position ist.
9. Schlossmechanismus nach Ansprüchen 6 - 8, wobei das Mittel zum Verhindern, dass der Riegel 3 in den Schlosskasten 1 gedrückt wird, der Zapfen 16 ist, der mit dem Zahnrad 15 verbunden ist und sich in

den Schlitz 20 erstreckt.

10. Schlossmechanismus nach Anspruch 9, wobei der Zapfen 16 eine Endposition hat, wenn der Riegel 3 voll aus dem Schlossmechanismus ausgefahren ist, wobei die Endposition auf oder über einer Linie liegt, die parallel oder im Wesentlichen parallel zu der Verschiebungsrichtung des Riegels 3 verläuft und durch die Drehungsmitte des Zahnrads 15 geht.
11. Schlossmechanismus nach einem der vorhergehenden Ansprüche, wobei die Position des gezahnten Elements 11 so in seinen beiden Endpositionen vorgespannt ist, dass das gezahnte Element 11 zwei stabile Positionen hat, wo das Eingriffsmittel 5 aus dem Aufnahmemittel 12 ausgerückt ist.

Revendications

1. Mécanisme de verrouillage pour une porte ou fenêtre, ledit mécanisme de verrouillage étant préparé pour recevoir un barillet de serrure (4) avec un moyen d'enclenchement (5) que l'on peut faire tourner de 360 degrés, ledit mécanisme de verrouillage comprenant :

- un élément denté (11) qui peut se déplacer le long d'une trajectoire courbe (13) et comporte un moyen de réception (12) préparé pour s'enclencher avec ledit moyen d'enclenchement (5), ladite trajectoire courbe (13) ayant une forme telle que, quand on fait tourner le moyen d'enclenchement (5) dans un sens, celui-ci s'enclenchera avec le moyen de réception (12) en un premier point sur ladite trajectoire courbe (13) et se désenclenchera du moyen de réception (12) en un second point sur ladite trajectoire courbe (13), de telle sorte que le moyen d'enclenchement (5) peut tourner de 360 degrés ;
- une roue dentée (15) engrenée avec ledit élément denté (11) et telle que le déplacement dudit élément denté (11) le long de ladite trajectoire courbe (13) entraîne une rotation de ladite roue dentée (15) ; et
- un pêne (3) pouvant être déplacé par un mouvement de rotation de ladite roue dentée (15),

caractérisé en ce que ledit élément denté est disposé de telle sorte que, pendant le fonctionnement normal du mécanisme de verrouillage, l'élément denté est situé entièrement d'un seul côté d'un plan disposé perpendiculairement au plan de symétrie du barillet de serrure et parallèlement à l'axe central de la clé du barillet de serrure et décalé de l'axe central de la clé du barillet de serrure de 16,5 mm.

2. Mécanisme de verrouillage selon la revendication 1,

dans lequel, quand on fait tourner le moyen d'enclenchement (5) dans le sens contraire, celui-ci s'enclenchera avec le moyen de réception (12) au second point sur ladite trajectoire courbe (13) et se désenclenchera du moyen de réception (12) au premier point sur ladite trajectoire courbe (13).

3. Mécanisme de verrouillage selon la revendication 1 ou 2, dans lequel ledit moyen d'enclenchement (5) est une protubérance et ledit moyen de réception (12) est un évidement dans l'élément denté (11). 5
4. Mécanisme de verrouillage selon l'une quelconque des revendications précédentes, dans lequel ledit l'élément denté (11) est disposé pour pouvoir coulisser le long de ladite trajectoire courbe (13). 10
5. Mécanisme de verrouillage selon l'une quelconque des revendications précédentes, dans lequel la partie de ladite trajectoire courbe (13) le long de laquelle le moyen d'enclenchement (5) sera enclenché avec le moyen de réception (12) a un rayon moyen supérieur au rayon de rotation dudit moyen d'enclenchement (5) et un centre géométrique différent du centre de rotation dudit moyen d'enclenchement (5). 15
6. Mécanisme de verrouillage selon l'une quelconque des revendications précédentes, dans lequel ladite roue dentée (15) est reliée à un ergot (16) qui s'étend perpendiculairement au plan de la roue dentée (15) pour entrer dans une fente (20) reliée audit pêne, de telle sorte que quand on fait tourner la roue dentée (15) le mouvement de rotation se traduit par un mouvement rectiligne ou sensiblement rectiligne dudit pêne (3). 20
7. Mécanisme de verrouillage selon la revendication 6, dans lequel ladite fente (20) s'étend dans le sens de sa longueur transversalement par rapport à la direction de déplacement dudit pêne (3). 25
8. Mécanisme de verrouillage selon l'une quelconque des revendications précédentes, dans lequel le mécanisme de verrouillage est équipé de moyens pour empêcher de repousser le pêne (3) dans le coffre (1) de la serrure quand le pêne (3) est dans sa position sortie. 30
9. Mécanisme de verrouillage selon les revendications 6 à 8, dans lequel ledit moyen pour empêcher de repousser le pêne (3) dans le coffre (1) de la serrure est l'ergot (16) relié à ladite roue dentée (15) et s'étendant dans ladite fente (20). 35
10. Mécanisme de verrouillage selon la revendication 9, dans lequel ledit ergot (16) a une position terminale quand le pêne (3) est en extension complète par rapport au mécanisme de verrouillage, ladite posi- 40

tion terminale étant placée sur ou au dessus d'une ligne qui est parallèle ou sensiblement parallèle à la direction de déplacement dudit pêne (3) et passe par le centre rotation de ladite roue dentée (15).

11. Mécanisme de verrouillage selon l'une quelconque des revendications précédentes, dans lequel ledit élément denté (11) est rappelé vers l'une ou l'autre de ses positions terminales, de telle sorte que ledit élément denté (11) présente deux positions stables dans lesquelles le moyen d'enclenchement (5) est désenclenché dudit moyen de réception (12). 45

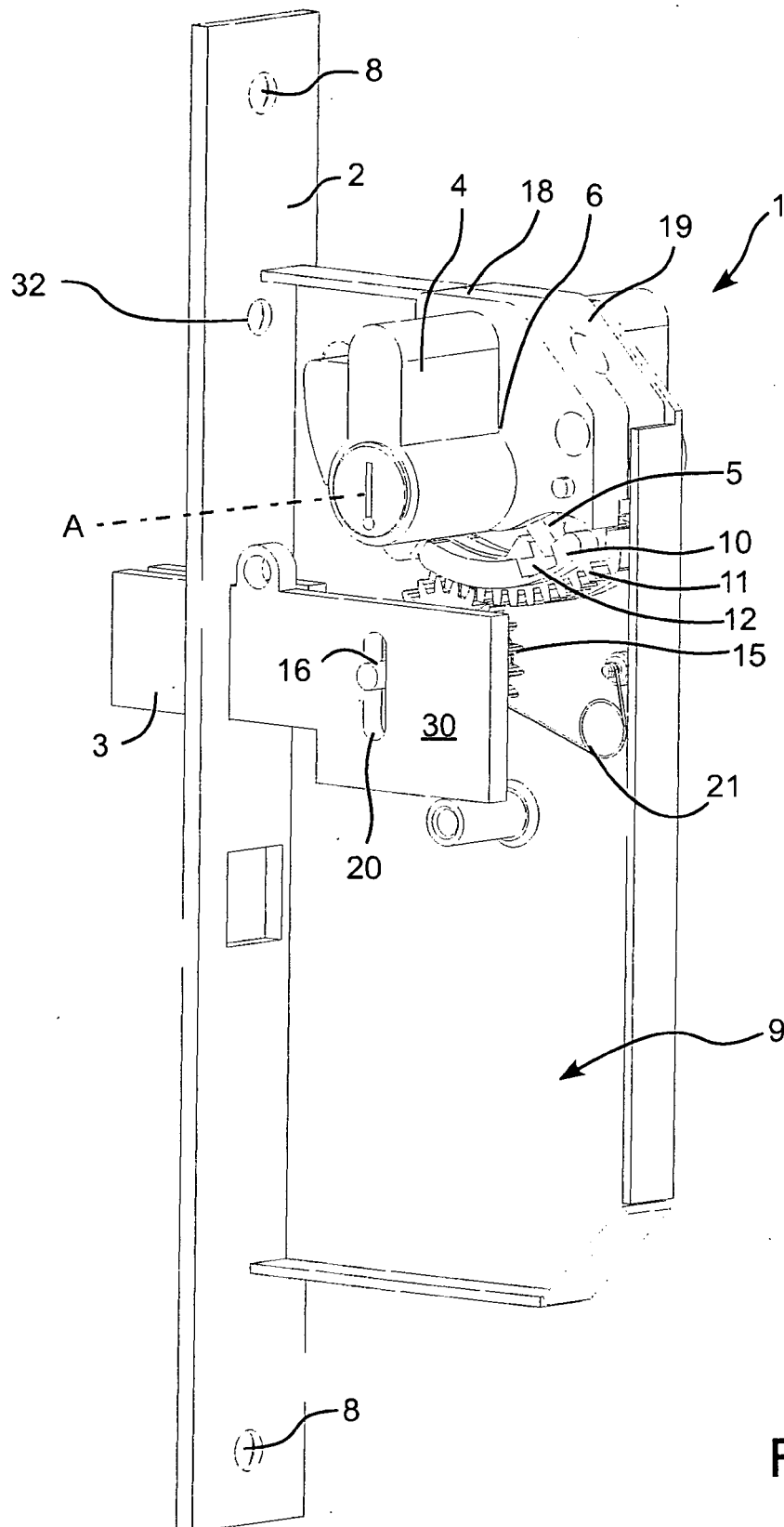


Fig. 1

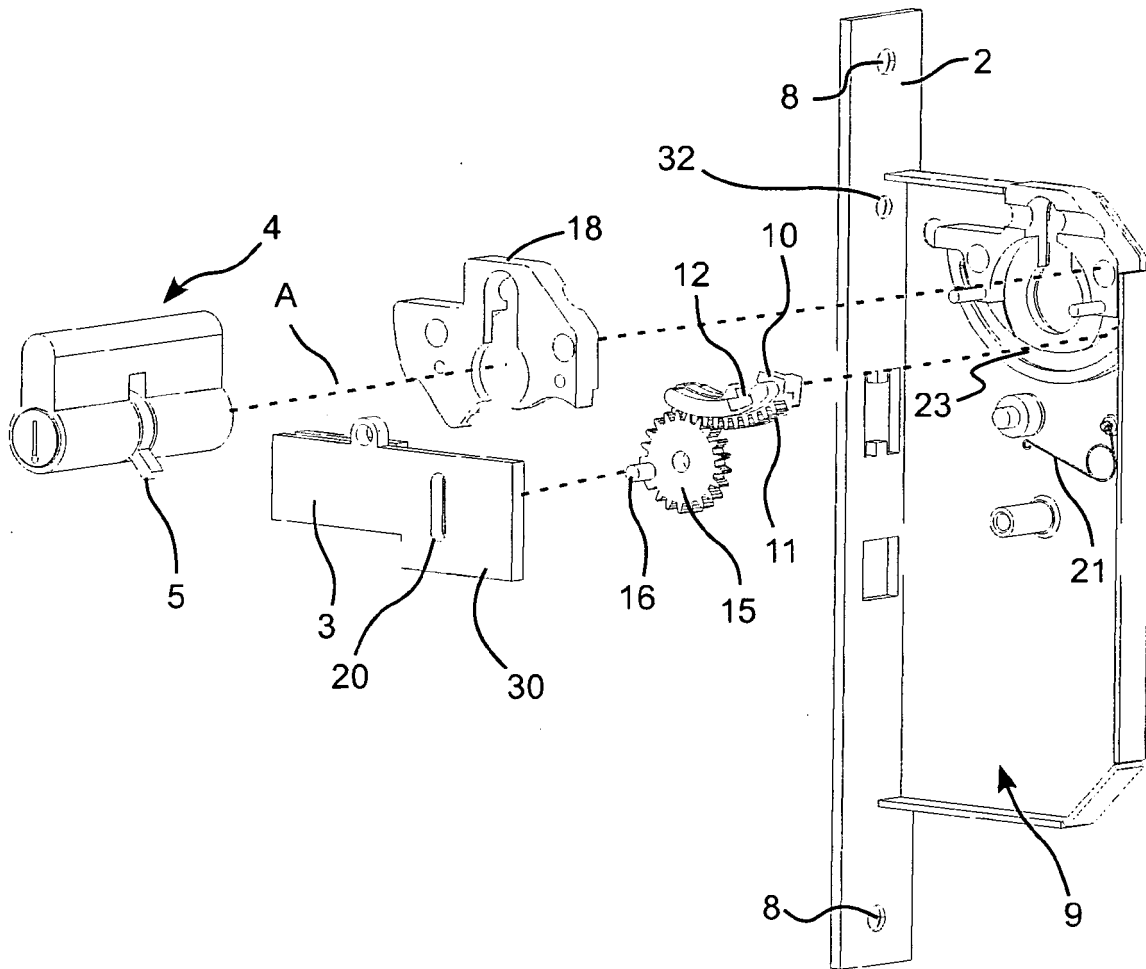


Fig. 2

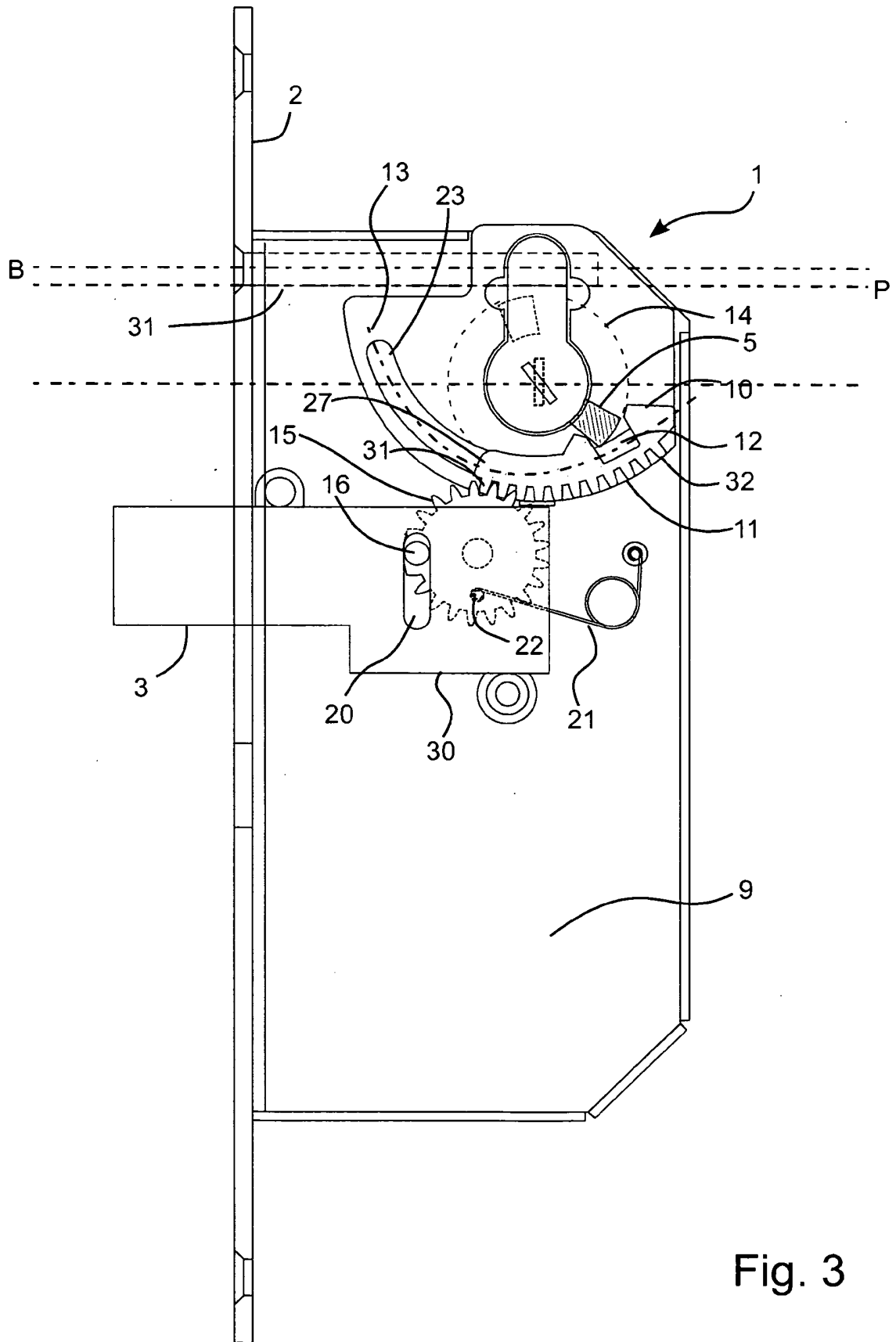


Fig. 3

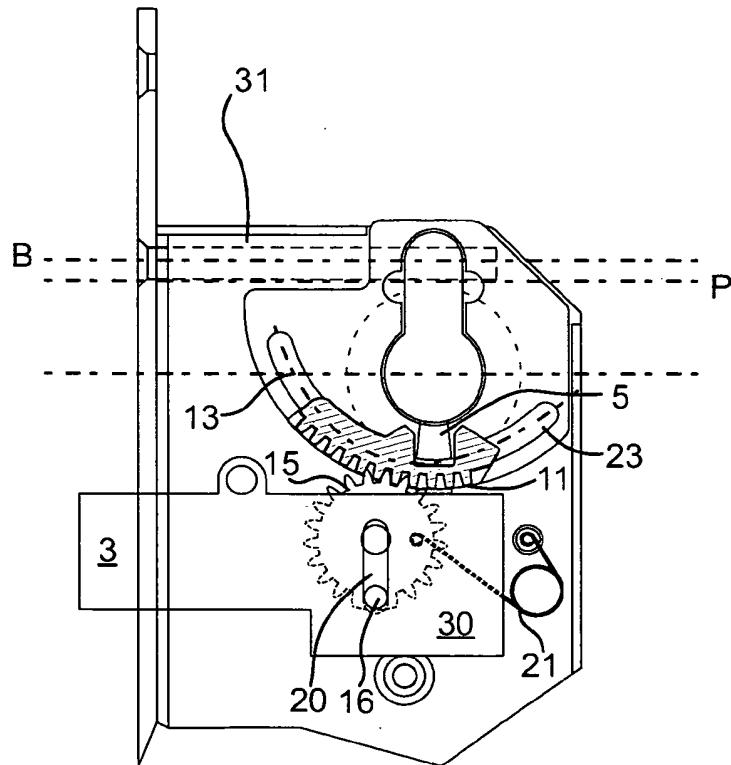


Fig. 4

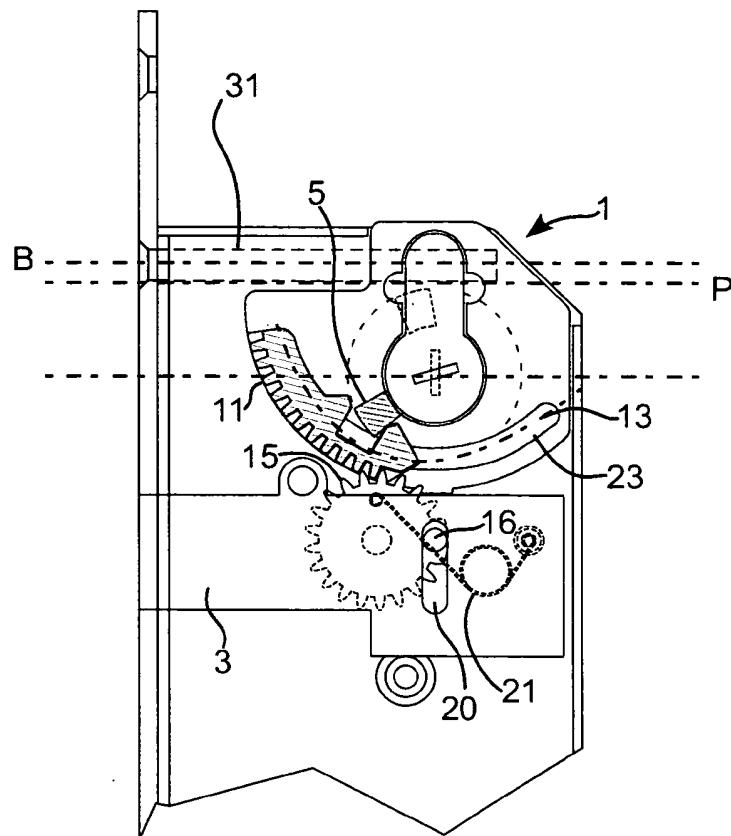


Fig. 5

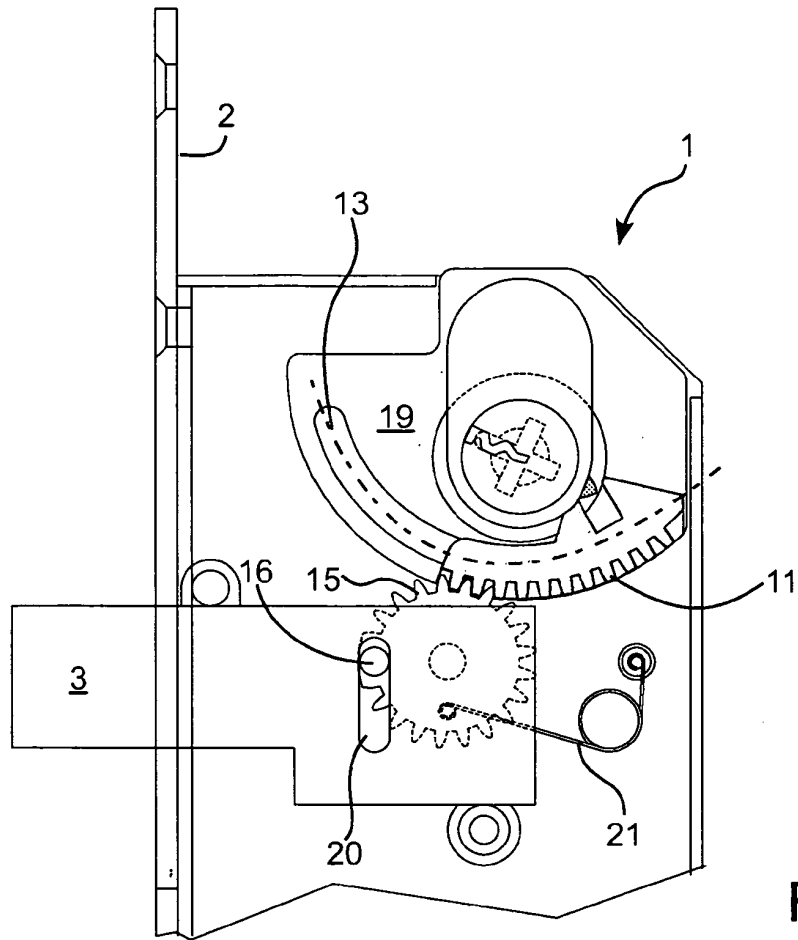


Fig. 6

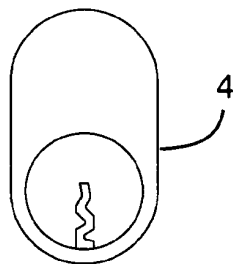


Fig. 7a

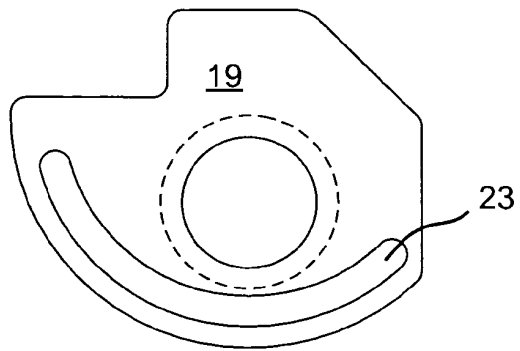


Fig. 7c

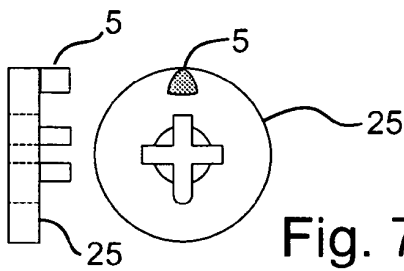


Fig. 7b

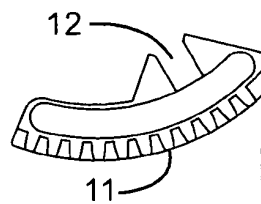


Fig. 7d

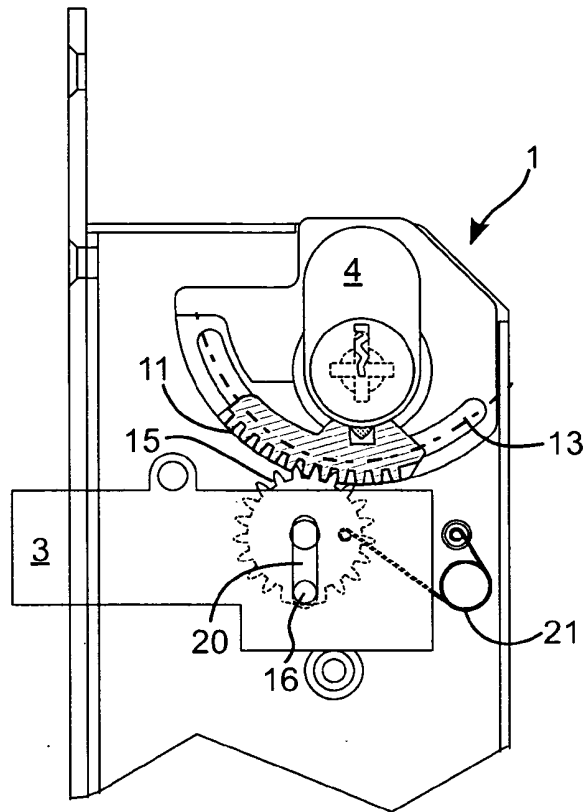


Fig. 8

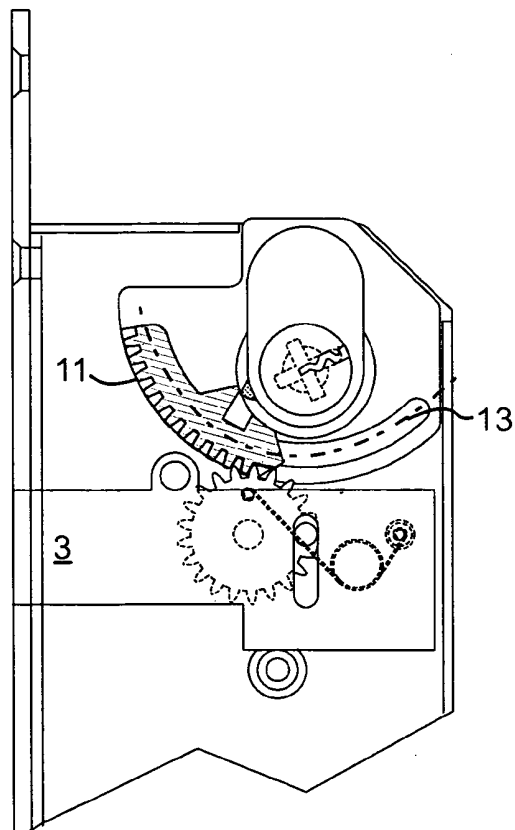


Fig. 9

REFERENCES CITED IN THE DESCRIPTION

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