United States Patent [19]

Townsend

[11] **3,874,348**

[45] **Apr. 1, 1975**

[54] **ROTARY INTERNAL COMBUSTION ENGINE**

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[73] Assignee: **Townsend Engineering Company,** Des Moines, Iowa

[22] Filed: **July 12, 1973**

[21] Appl. No.: **378,701**

Related U.S. Application Data

[63] Continuation-in-part of Ser. Nos. 286,189, Sept. 5, 1972, abandoned, and Ser. No. 301,096, Oct. 26, 1972, Pat. No. 3,857,372.

[52] **U.S. Cl.** ..... **123/44 B**

[51] **Int. Cl.** **F02b 57/00**

[58] **Field of Search** ..... 123/44 B

[56] **References Cited**

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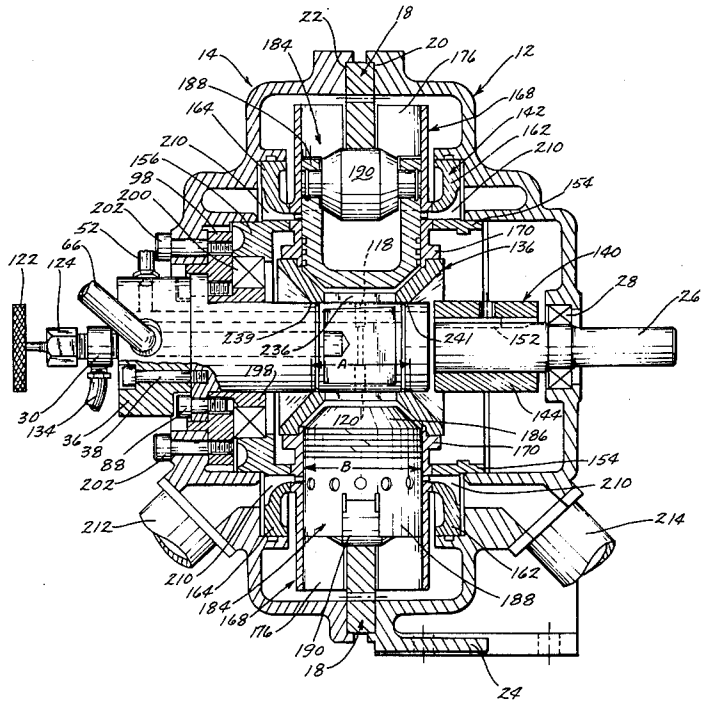
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Primary Examiner—C. J. Husar
Attorney, Agent, or Firm—Zarley, McKee, Thomte & Voorhees

[57] **ABSTRACT**

A rotary internal combustion engine is disclosed and generally comprises a rotor means which is rotatably mounted on a stationary core mounted in an engine frame. The rotor means has a plurality of cylinders spaced radially thereon. A free floating piston is slidably mounted in each of the cylinders with the heads of the pistons being positioned towards the center of the rotor. A roller is mounted in the skirt end of the piston which rides against a circular cam. The centrifugal force created by the rotation of the rotor causes the pistons to follow the internal cam surface of the cam during rotation. The inner end of the core is provided with fuel and air ports adapted to communicate with the cylinders as the rotor rotates with respect to the core. An air passageway is formed in the core and is in communication with the air port and an air pump to provide a supply of air for purging, cooling and charging the cylinders. A fuel passageway is provided in the core and is in communication with the fuel ports and a supply of fuel to supply fuel under pressure to the fuel ports. An adjustable needle valve is provided in the fuel passageway to permit the precise adjustment or metering of the fuel being supplied to the fuel ports.

4 Claims, 20 Drawing Figures



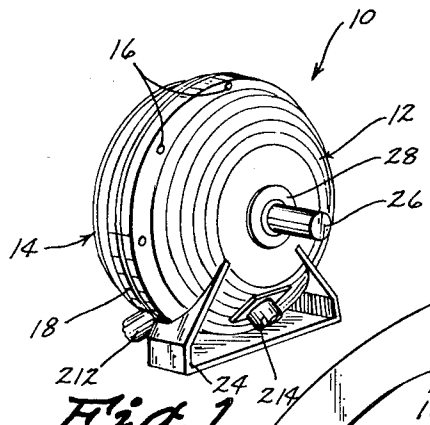


Fig. 1

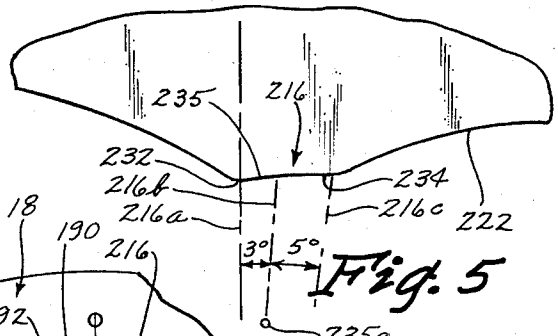


Fig. 5

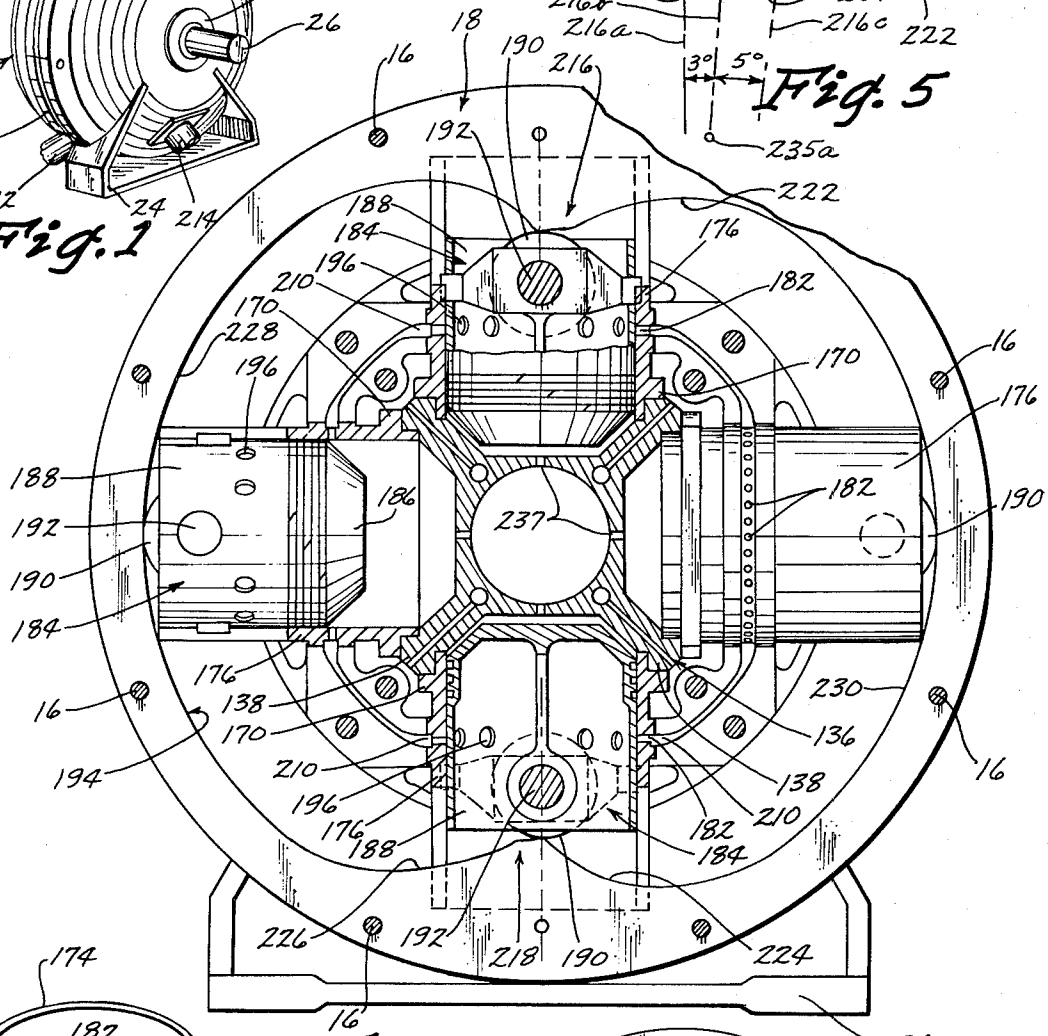


Fig. 2

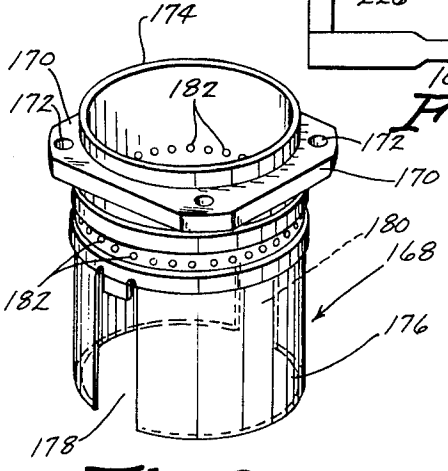


Fig. 3

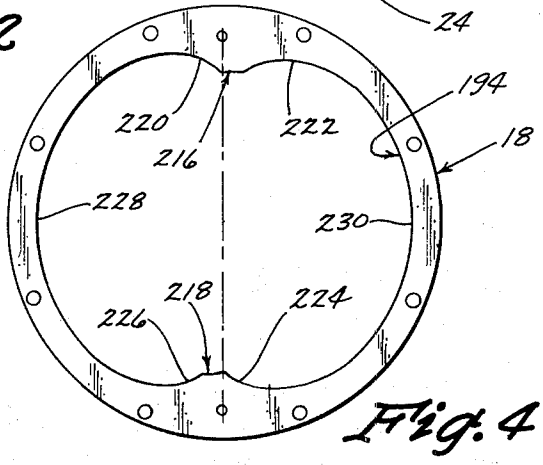


Fig. 4

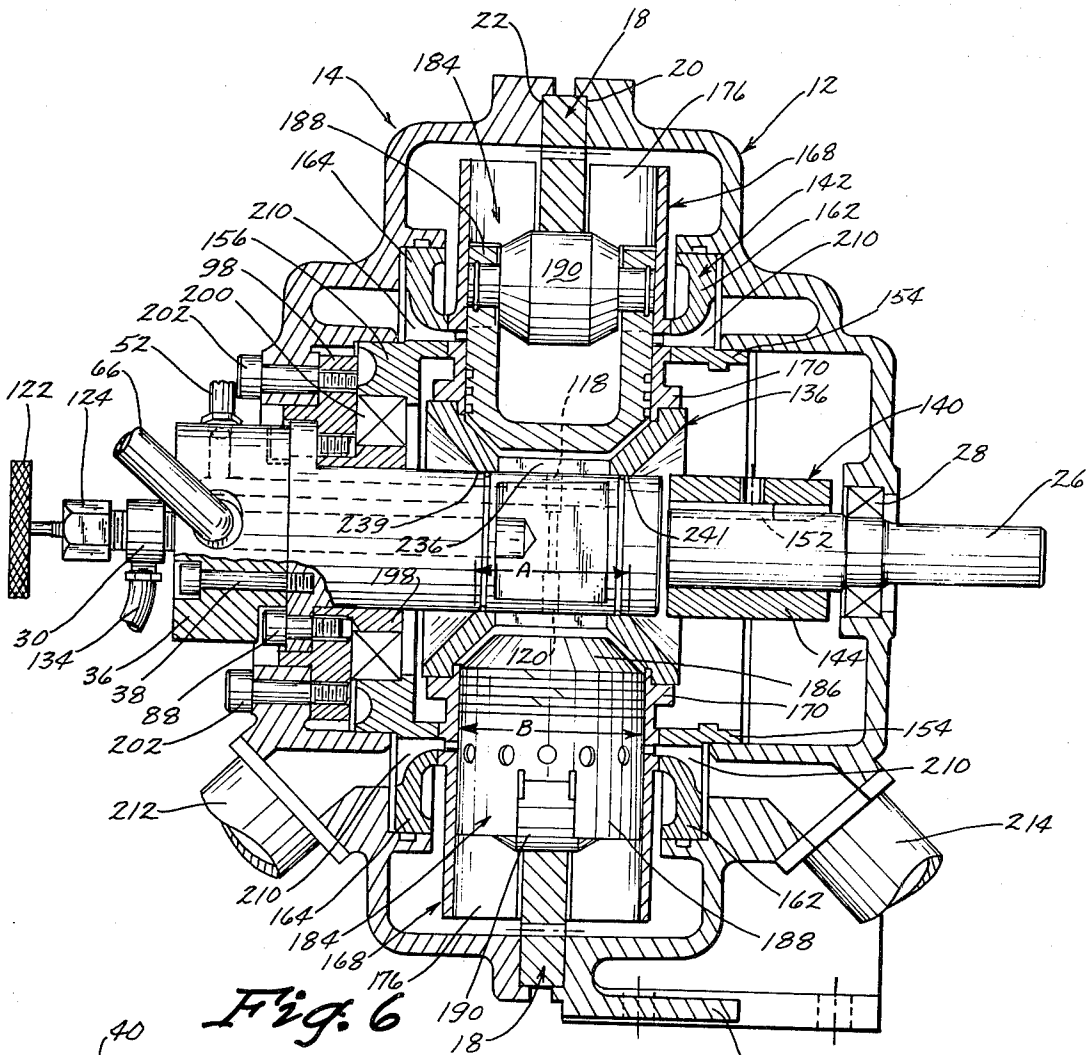

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Fig. 6

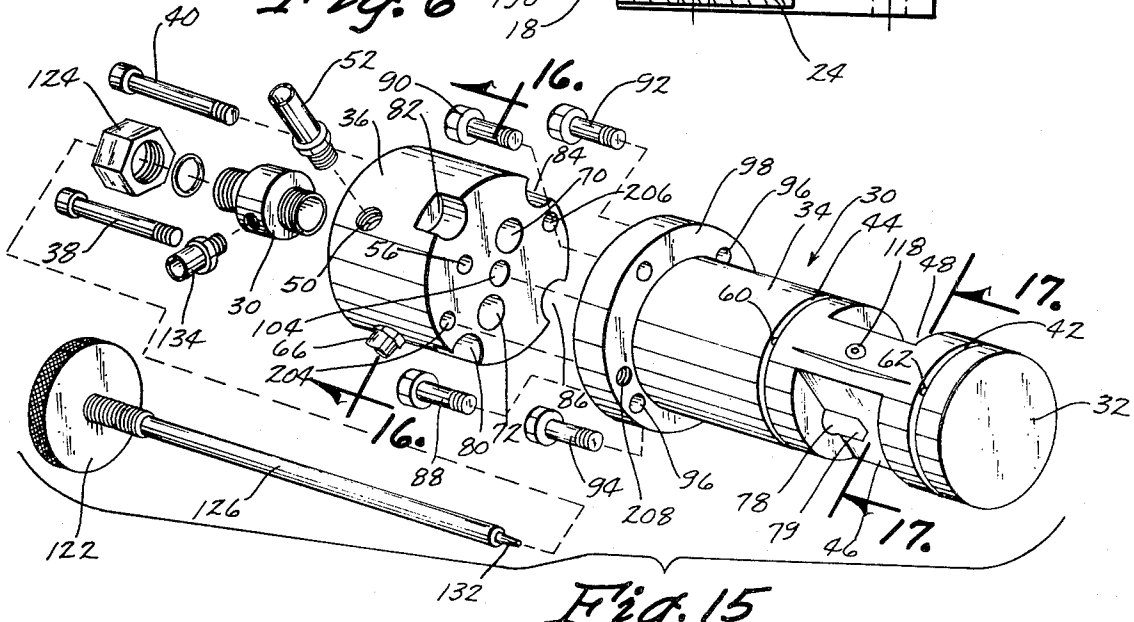

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Fig. 15

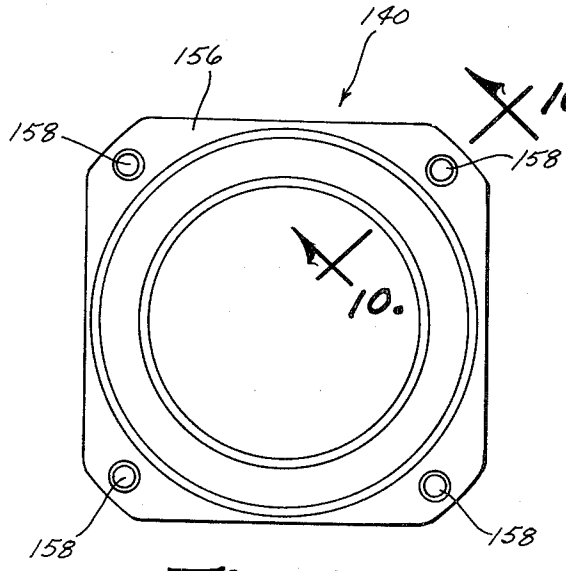


Fig. 7

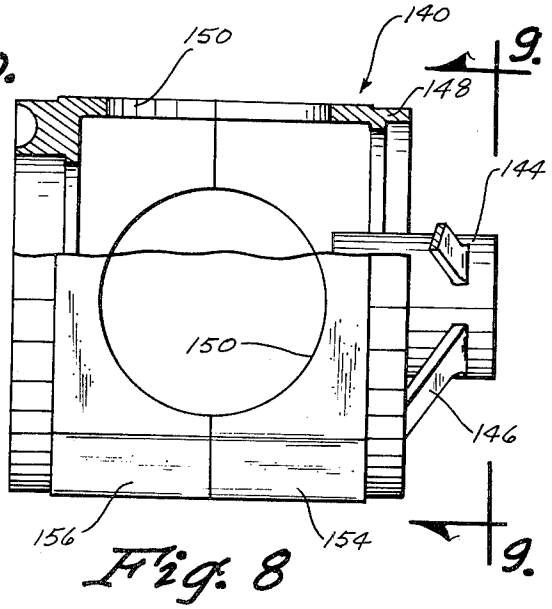


Fig. 8

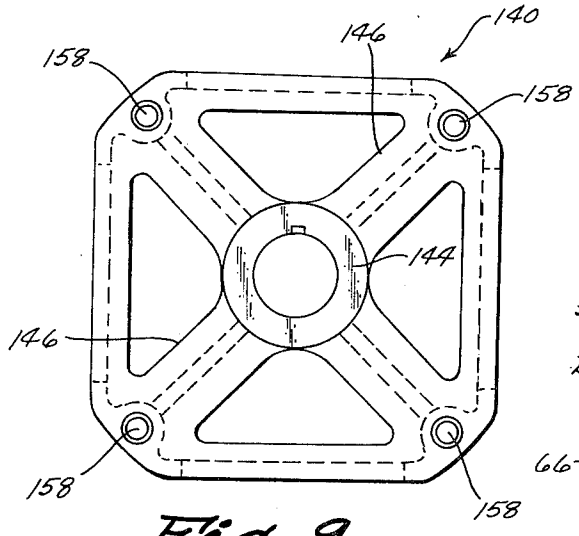


Fig. 9

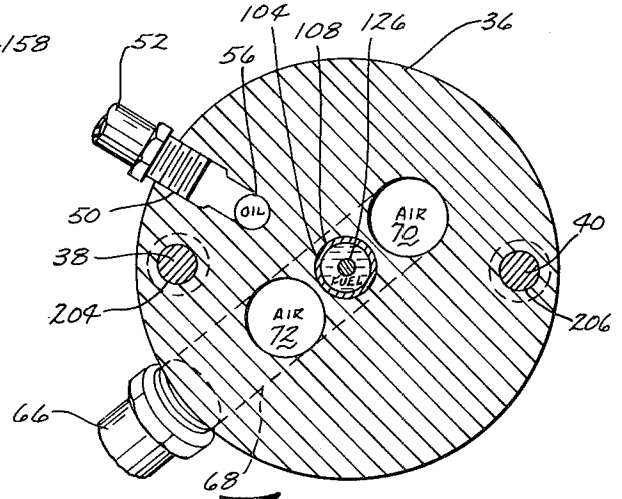


Fig. 16

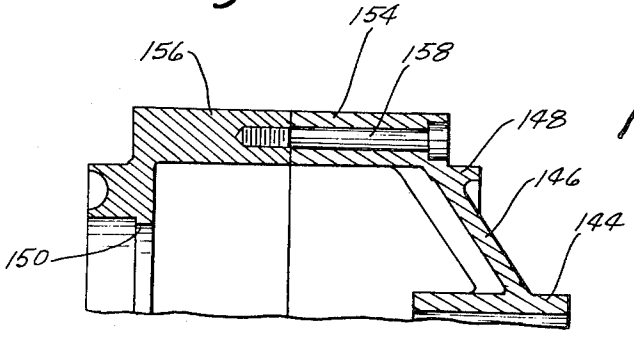


Fig. 10

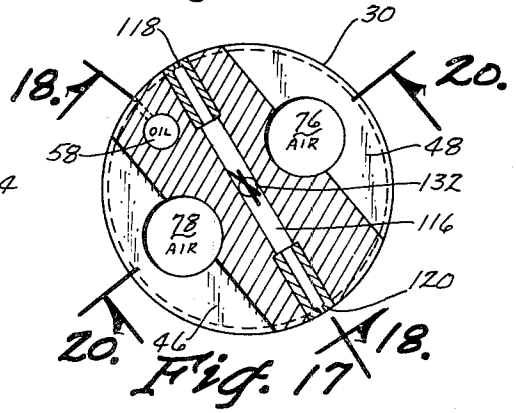


Fig. 17

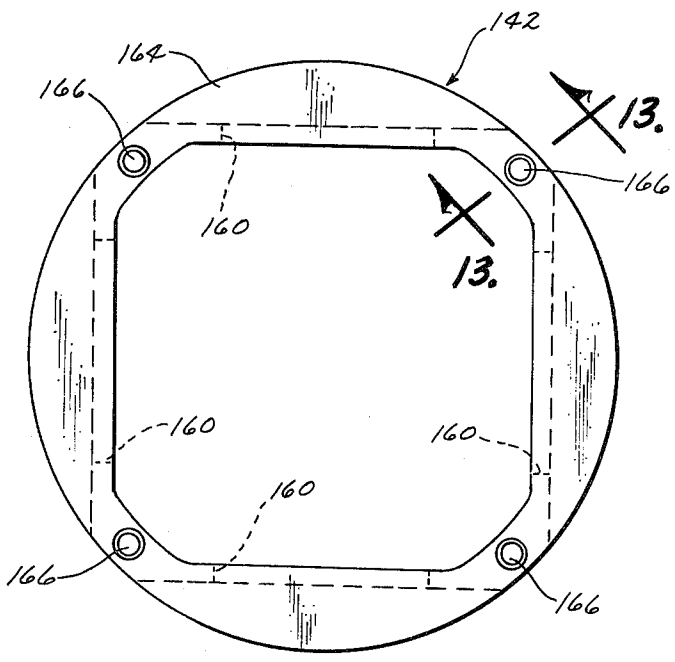


Fig. 11

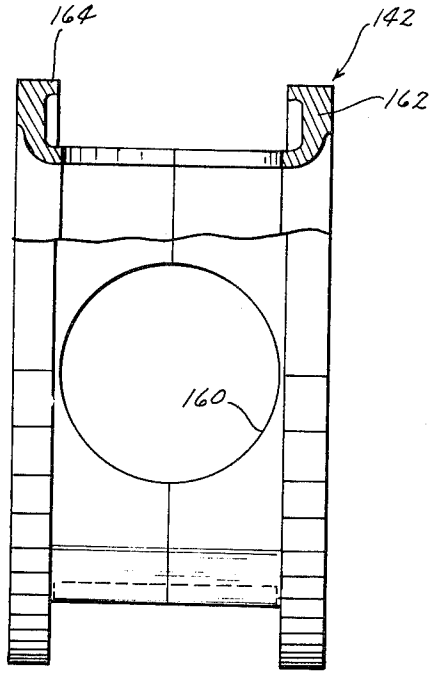


Fig. 12

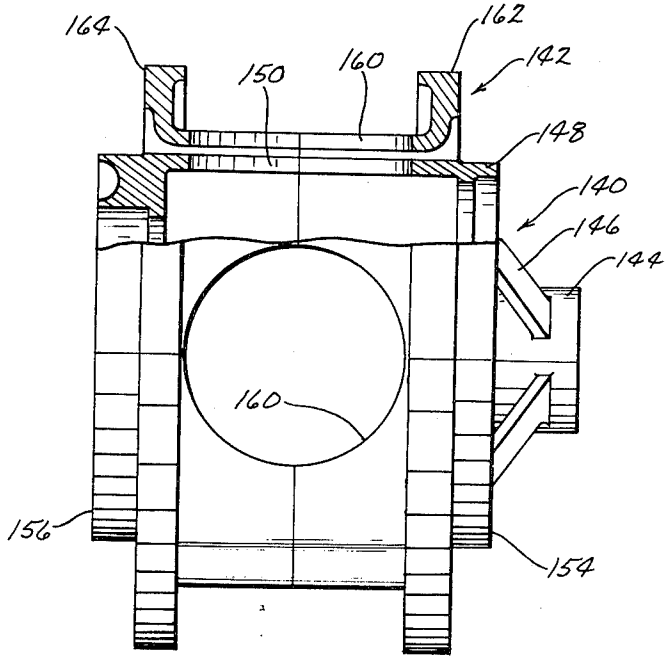


Fig. 14

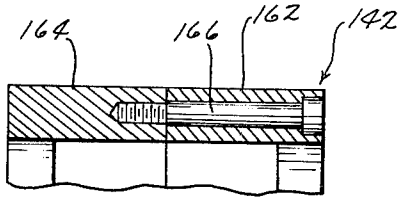


Fig. 13

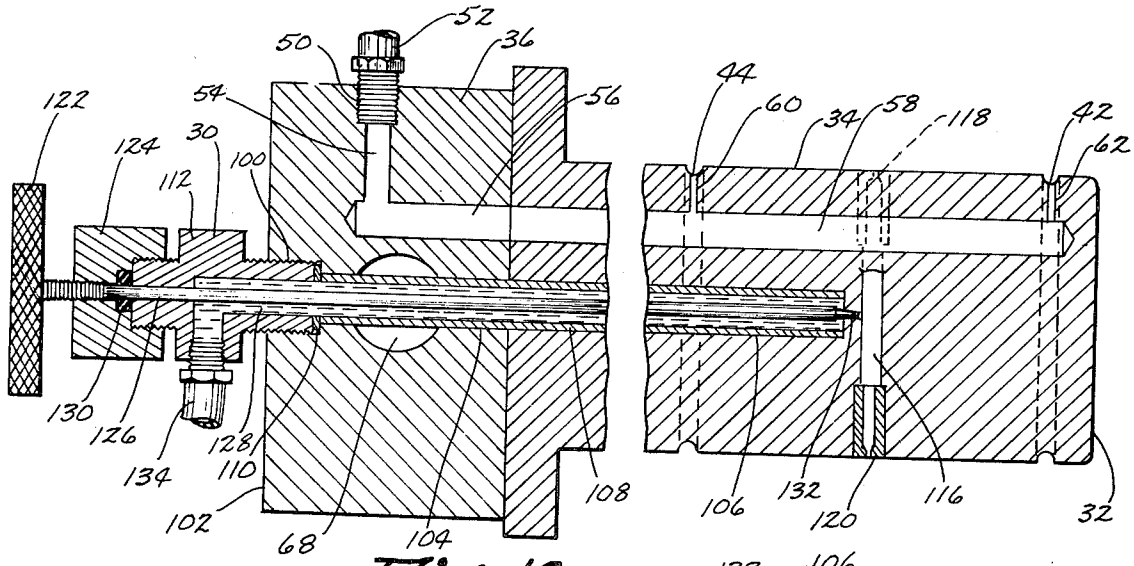


Fig. 18

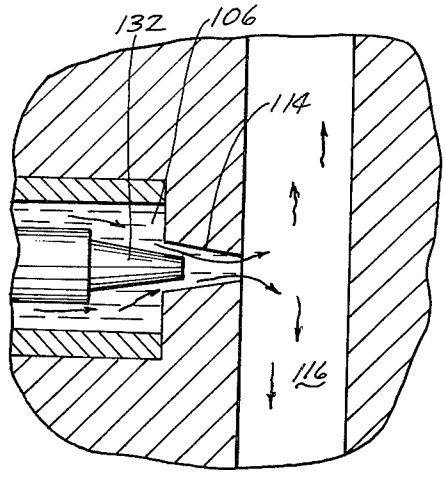


Fig. 19

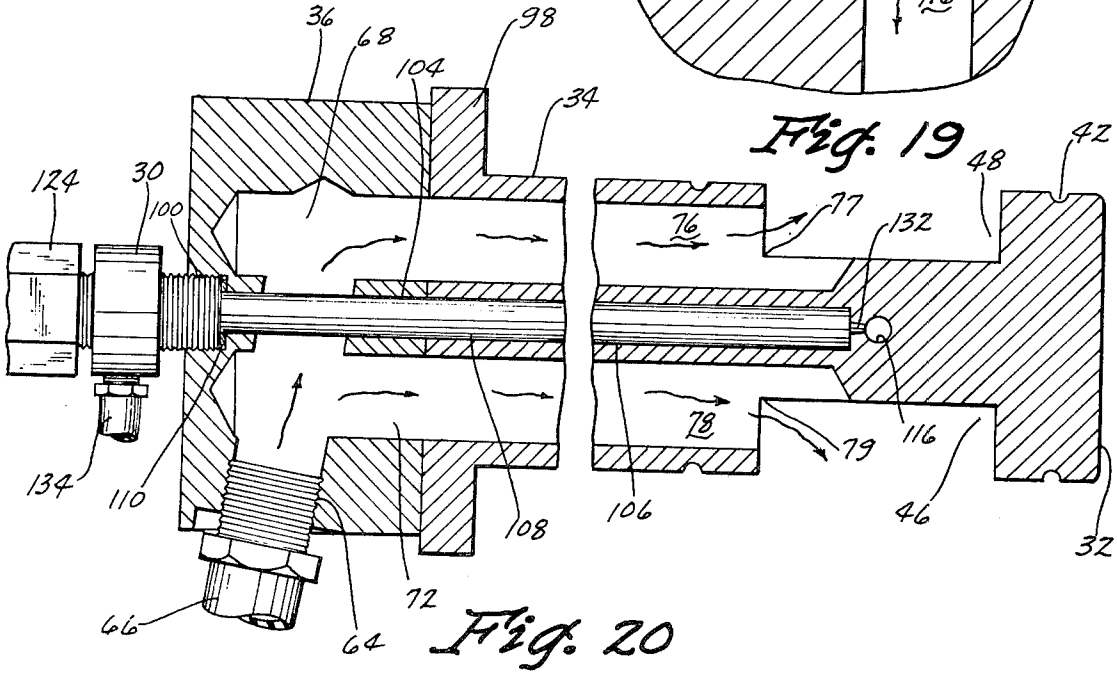


Fig. 20

ROTARY INTERNAL COMBUSTION ENGINE

This application is a continuation-in-part application of co-pending applications, Ser. No. 286,189 filed Sept. 5, 1972, now abandoned and Ser. No. 301,096 filed Oct. 26, 1972, now U.S. Pat. No. 3,857,372.

BACKGROUND OF THE INVENTION

In the usual internal combustion engines, the heat generated by the explosion is partly dissipated or conducted into the walls of the surrounding metals, and as it is conducted through the walls, it is radiated off to the air from fins on the outer surface of the cylinders, or is conducted into a liquid coolant in a jacket around the cylinders.

In rotary internal combustion engines of the cam type, the rollers of the pistons are forced outwardly into engagement with the cam plates so that the rollers will follow the shape of the cam plate. Frequently, the centrifugal force is not enough to overcome the vacuum effect when the pistons are required to suck in their charge of air in conventional fashion.

In a conventional engine of either the compression or spark ignition type, there is a problem of detonation and/or pre-ignition caused by the sudden pressure and temperature rise immediately after ignition while the crank is still near the top center where there is very little volume into which the gas can expand. To cope with this problem, a combustion deterrent such as lead has been used to slow the burning of the fuel, or the fuel has been slowly injected into the cylinder as in diesel engines so that the combustion will take place slowly as the piston expands down into the cylinder. This solution requires very careful timing and control and it is inefficient since some of the burning is retarded until the piston has been considerably moved towards an expanded position.

Conventional rotary internal combustion engines have experienced some problems in the manner of sealing the respective cylinders, air ports, gas ports, etc. Elaborate sealing mechanisms have been attempted but the problem is largely unsolved. A further problem exisiting in present rotary internal combustion engines is in supplying air and fuel to the respective cylinders at the proper time in an efficient manner.

Therefore, a principal object of the invention is to provide an unique method and means of cooling an internal combustion engine.

A further object of the invention is to provide a rotary internal combustion engine in which the exhaust is passed out of the individual cylinders instead of through a common valve port.

A further object of the invention is to provide a method and means for preventing excessive pressure and temperature from occurring in the cylinders during combustion.

A further object of the invention is to provide a rotary internal combustion engine wherein the pistons are maintained in a partially expanded position immediately after maximum compression to permit complete combustion without excessive pressures and temperatures being experienced.

A further object of the invention is to provide a rotary internal combustion engine comprising a stationary core having the rotor rotatably mounted thereon.

A further object of the invention is to provide a rotary internal combustion engine including means for supplying air thereto for purging, cooling and recharging the cylinders.

A further object of the invention is to provide a rotary internal combustion engine wherein air under pressure is employed to force the pistons outwardly into engagement with the cam plate.

A further object of the invention is to provide a rotary internal combustion engine having improved efficiency.

A further object of the invention is to provide a rotary internal combustion engine which is economical to manufacture and durable in use.

These and other objects will be apparent to those skilled in the art.

SUMMARY OF THE INVENTION

In the instant invention, a stationary core is provided in the engine frame upon which the rotor rotates. A plurality of radilly spaced cylinders are mounted on the rotor with a piston being movably mounted in each of the cylinders. A cam plate is provided on the frame and has a cam surface thereon which is engaged by the rollers secured to each of the pistons so that the pistons will be moved between positions of compression and expansion as the rotor is being rotated with respect to the core. A rotatable shaft is secured to the rotor and extends outwardly from the frame.

A fuel passageway extends through the core for supplying fuel to the respective cylinders. A needle valve is positioned in the fuel passageway to permit the precise control of the fuel being supplied to the cylinders. An air passageway extends through the core for supplying air to the cylinders for charging, cooling and purging the cylinders. An oil passageway extends through the core for supplying oil to a pair of spaced apart annular grooves formed in the periphery of the core to provide a seal between the core and the rotor. The oil also serves to lubricate the rotor and core.

The preferred embodiment of the cam plate comprises a pair of lobes positioned **180°** apart on the cam plate. A slightly curved cam surface is provided on each of the lobes adjacent the position of maximum compression so that the pistons may partially move to an expanded position immediately after the point of maximum compression to achieve substantially complete combustion without causing excessive pressure and temperature within the cylinder.

BRIEF DESCRIPTION OF THE DRAWINGS

This invention consists in the construction, arrangements and combination of the various parts of the device, whereby the objects contemplated are attained as hereinafter more fully set forth, specifically pointed out in the claims, and illustrated in the accompanying drawings, in which:

FIG. **1** is a perspective view of the engine of this invention;

FIG. **2** is an enlarged sectional view seen along lines **2 — 2** of FIG. **1** with portions thereof cut-away to more fully illustrate the invention:

FIG. **3** is a perspective view of one of the cylinders of the engine:

FIG. **4** is a plan view of the cam plate employed in the engine:

FIG. **5** is an enlarged partial plan view of the cam plate of FIG. **4**:

FIG. 6 is a sectional view of the engine as would be seen along lines transverse to the sectional view of FIG. 2:

FIG. 7 is an end view of one of the rotor members:

FIG. 8 is a partial top view of the rotor of FIG. 7:

FIG. 9 is an end view of the rotor member as seen along lines 9 — 9 of FIG. 8:

FIG. 10 is an enlarged sectional view seen along lines 10 — 10 of FIG. 7:

FIG. 11 is a plan view of another of the rotor members:

FIG. 12 is an end view of the rotor member of FIG. 11:

FIG. 13 is an enlarged sectional view seen along lines 13 — 13 of FIG. 11:

FIG. 14 is a top view of the rotor members with portions thereof cutaway to more fully illustrate the invention:

FIG. 15 is an exploded perspective view of the core of the engine:

FIG. 16 is an enlarged sectional view seen along lines 16 — 16 of FIG. 15:

FIG. 17 is an enlarged sectional view seen along lines 17 — 17 of FIG. 15:

FIG. 18 is an enlarged partial sectional view seen along lines 18 — 18 of FIG. 17:

FIG. 19 is an enlarged partial sectional view seen along lines 18 — 18 of FIG. 17; and

FIG. 20 is an enlarged sectional view seen along lines 20 — 20 of FIG. 17.

DESCRIPTION OF THE PREFERRED EMBODIMENTS

The engine of this invention is designated by the reference numeral **10** and generally comprises engine frames **12** and **14** secured together by bolts **16** or other suitable means as seen in FIGS. **1** and **2.** As shown in FIG. **1,** a circular cam plate **18** is positioned between the frames **12** and **14** with the bolts **16** extending therethrough. Cam plate **18** is positioned with respect to the peripheral edges of frames **12** and **14** by the annular recesses **20** and **22** formed therein respectively (FIG. **6**). Frame **12** includes a bracket portion **24** extending therefrom for mounting the engine.

Drive or rotor shaft **26** rotatably extends inwardly through the frame **12** and is supported therein by a main bearing **28.** The numeral **30** refers generally to a core which extends through frame **14** into engine **10** so that its inner end **32** is positioned closely adjacent the inner end of the shaft **26.** Core **30** is comprised of core members **34** and **36** which are secured together by a pair of bolts **38** and **40.** Core member **34** is provided with a pair of spaced apart annular oil grooves **42** and **44** formed in the peripheral surface thereof as seen in FIG. **15.** A pair of cut-away areas or notches **46** and **48** are also formed in core member **34** on opposite sides thereof as also seen in FIG. **15.**

Core member **36** is provided with an internally threaded port **50** having an oil fitting or conduit **52** mounted therein. Port **50** communicates with a bore **54** extending radially inwardly therefrom as seen in FIG. **18.** The inner end of bore **54** communicates with a longitudinally extending bore **56** which communicates with a longitudinally extending bore **58** formed in core member **34.** A radially extending oil passageway **60** extends from bore **58** and communicates with oil groove **44.** An oil passageway **62** extends from bore **58** and communicates with the oil groove **42.** Oil under pressure is supplied through the oil conduit **52** so that oil will be supplied to the oil grooves **42** and **44** to lubricate the rotor which is rotatably mounted thereon as will be explained in more detail hereinafter. The oil in the grooves **42** and **44** also serves to provide a seal between the rotor and the core as also will be described hereinafter.

Core member **36** is provided with an internally threaded port **64** having an air line **66** threadably mounted therein. Air line **66** is in communication with a source of air under pressure. Port **64** communicates with an air compartment **68** having a pair of spaced apart air passageways or bores **70** and **72** extending longitudinally therefrom. Core member **34** is provided with a pair of elongated air passageways or bores **76** and **78** which communicate with the bores **70** and **72.** As seen in FIG. **20,** the inner ends of the bores **76** and **78** communicate with the notches **48** and **46** respectively to provide air ports **77** and **79** respectively. As seen in FIG. **15,** four notches, **80, 82, 84** and **86** are formed in the exterior surface of core member **36** adjacent the inner end thereof which are adapted to receive the head portion of the bolts **88, 90, 92** and **94** respectively which extend through openings **96** formed in the flange portion **98** of core member **34.**

An internally threaded bore **100** is formed in the outer end **102** of core member **36.** An elongated bore **104** extends longitudinally through the core member **36** from the inner end of the bore **100** and communicates with an elongated longitudinally extending bore **106** formed in core member **34.** The numeral **108** refers to an elongated metal tube which is positioned in the bores **104** and **106** as seen in FIG. **18.** Tube **108** is provided with a flat head portion **110** which engages the inner end of a fitting **112** threadably mounted in the bore **100.** A tapered valve seat or opening **114** is provided in the core member **36** at the inner end of bore **106** which communicates with a transversely extending bore or passageway **116.** Bore **116** extends radially outwardly to the periphery of core member **34** and has a pair of replaceable fuel jets **118** and **120** mounted in the outer ends thereof respectfully.

The numeral **122** refers to a manually adjustable needle valve which is threadably mounted in the collar **124** which is threadably secured to the outer end of the fitting **112.** Needle valve **122** rotatably extends through a bore **126** formed in fitting **112,** through bore **128** formed in fitting **112** and through the elongated tube **106.** Suitable seal means **130** embraces the needle valve **122** as also illustrated in FIG. **18.** The inner end of needle valve **122** is provided with a tapered head portion **132** which is complementary to the valve seat **114** to permit the valve seat **114** to be selectively opened and closed through the rotational movement of the needle valve **122.** Fuel line **134** is threadably connected to the fitting **112** and communicates with the bore **128** to supply fuel thereto and to the interior of the tube **106,** valve seat **114** and bore **116.** Fuel line **134** is in communication with a source of suitable fuel under pressure.

The numeral **136** refers to a rotor means rotatably mounted on core member **34** and comprising rotor members **138, 140** and **142.** Rotor member **140** comprises a hub portion **144** and spokes **146** which extend radially outwardly therefrom to a rim portion **148.** Rim portion **148** is provided with four circular openings **150**

formed therein. Hub portion **144** is secured to shaft **126** by any suitable means such as a key **152** or the like. Rim portion **148** is comprised of rim members **154** and **156** held together by bolts **158.** Rotor member **142** embraces rotor member **140** and has four circular openings **160** formed therein which register with the openings **150** in rotor member **140.** Rotor member **142** comprises a pair of members **162** and **164** secured together by bolts **166.** The numeral **168** refers to a cylinder which is mounted in each of the openings **160** and having a flange portion **170** engaging rotor member **138** as illustrated in FIG. **6.** Each of the cylinders **168** is secured to the rotor member **138** by means of screws or the like extending through the openings **172** formed in flange **170** and being received by the rotor member **138.**

Each of the cylinders **168** generally comprises an inner end portion **174** and a skirt portion **176.** Skirt portion **176** is provided with opposing slots **178** and **180** formed therein and a plurality of exhaust openings **182** which extend through the cylinder around the circumference thereof. A piston **184** is slidably mounted in each of the cylinders **168** and generally comprises a head portion **186** and a skirt portion **188.** A roller **190** is mounted on shaft **192** which is secured to skirt portion **188.** Roller **190** rolls upon the cam surface **194** of the cam plate **18** to cause the piston to move with respect to the cylinder as the rotor of the engine rotates. The configuration of the cam plate will be explained in more detail hereinafter. Each of the pistons is provided with a plurality of openings **196** formed in the skirt portion and extending therethrough.

The numeral **198** refers to a bushing which embraces core member **34** and which supports a bearing **200.** As seen in FIG. **6,** the rim portion **156** of rotor member **144** engages and is supported by the bearing **200.** The inner ends of bolts **88, 90, 92** and **94** are threadably secured to the bushing **198** as also illustrated in FIG. **6.** Bolts **202** extend through frame **14** and are threadably secured to the bushing **198.** Bolts **38** and **40** extend through the openings **204** and **206** of core member **36** and are threadably received by the threaded openings **208** in flange **98** of core member **34.**

An exhaust chamber **210** is provided in the engine and extends around each of the cylinders as illustrated in FIG. **6.** The exhaust chamber **210** is in communication with a pair of exhaust pipes **212** and **214.** The exhaust chamber **210** is in communication with the exhaust openings **182** in the cylinders **168.**

Cam surface **194** of cam plate **18** includes oppositely disposed lobes **216** and **218.** Each of the lobes are provided with a "dwell" area as will be explained in more detail hereinafter. For purposes of description, the cam surfaces closely adjacent the opposite sides of lobe **216** will be indicated by the reference numerals **220** and **222** respectively. The cam surfaces closely adjacent the opposite side of lobe **218** will be indicated by the reference numerals **224** and **226** respectively. The cam surface approximate midway between **226** and **220** will be referred to by the reference numeral **228** while the cam surface approximately midway between **222** and **224** will be referred to by the reference numeral **230.**

FIG. **5** illustrates lobe **216** in somewhat greater detail. The maximum projecting portion of lobe **216** will be referred to by the reference numeral **232.** The engagement of the piston roller with **232** causes the piston to be moved inwardly with respect to its cylinder to achieve maximum compression at the top center. As seen in FIG. **5,** the lobe **216** is provided with a "dwell" portion **234** which has a radius measured from the geometric center of cam plate **18,** whereupon the piston will be maintained in a stationary position. Portion **234** spans a distance of about 5° and has a typical radius of 4.75 inches. Cam portion **236** connects the maximum projecting portion **232** with portion **234.** Portion **235** spans a distance of 3° and has a typical radius of 1 inch. Portion **235** permits the piston to be moved slightly to an expanded position for a period after maximum compression as will be explained in more detail hereinafter. Inasmuch as lobes **216** and **218** are identical, lobe **218** will not be described in detail. The lines **216***a*, **216***b* and **216***c* are radius lines extending from the geometric center of the cam plate **18.** Lines **216***a* and **216***b* span **3°** and define cam portion **235.** Lines **216**b and **216**c span **5°** and define cam portion **234.** The center **235**c of the arc of portion **235** appears on line **216***b*.

In operation, fuel under pressure is constantly supplied to the interior of the tube **106** and the bore **116.** The only time that fuel will pass from the jets **118** and **120** is when the jets communicate with the open inner ends of the cylinders **168** by way of the ports **237** formed in rotor member **138.** When the fuel jets **118** and **120** do communicate with the cylinders **168,** fuel will be sprayed into the interior of the cylinders. Air under pressure is constantly supplied to the air chamber **68** so that air will be discharged from the inner ends of bores **76** and **78** (ports **77** and **79**) when the notches **46** and **48** communicate with the interiors of the respective cylinders through the ports **237** as the rotor is being rotated around the stationary core **30.** Oil under pressure is also supplied to the oil grooves **42** and **44** as previously described so that lubricating oil is provided between the inside surface of the annular shoulders **239** and **241** of rotor element **138** and the exterior surface of core member **34.** The film of oil between the core member **34** and the rotor member **138** also serves to seal the ports **237** with respect to the area outwardly of the rotor.

FIG. **2** illustrates the top and bottom pistons in firing positions. In the position of FIG. **2,** the rollers on the pistons are on the top dead center areas **232** of the lobes **216** and **218** respectively. Inasmuch as the operation of the top piston is identical to the operation of the bottom piston (FIG. **2**), only the operation or cycles of the top piston will be described. Assuming that the air and fuel have been supplied to the interior of the cylinder, the shape of the cam compresses the fuel and air mixture to the maximum compression ratio at top dead center, at which time ignition is started. Before the pressure in the cylinder can build up beyond desired limits, the piston is permitted to expand slightly to its dwell position (**234**) and is held in the same relative position of expansion until combustion has been completed. During the dwell period, the combustion builds the pressure up to the maximum pressure possible with consumption of the total charge but the pressure is not above desired limits because the volume of the combustion chamber has been predetermined by the dwell in the cam. After complete combustion and substantially all of the chemical energy of the fuel has been converted to heat, the piston is allowed to expand and to convert that heat energy into mechanical energy, or torque, but not until all of the combustion is complete and the entire force of the combustion is available at

the beginning of the expansion stroke, after the dwell.

The slight dwell after top dead center provides many advantages. First, it eliminates the possibility of peak pressures rising too high because as long as the ignition timing is late enough to allow the piston to expand to its dwell volume, the combustion can never build up to a pressure higher than is possible in that dwell volume. Secondly, the accuracy necessary for ignition timing is much less because within extreme limits, it is only necessary to time the ignition late enough to allow the piston to get to its dwell position ahead of the combustion. If the ignition is later than desired and the piston gets to its dwell position farther ahead of the ignition than is necessary, there is no great loss because the combustion will catch up while the piston is dwelling. The third advantage is the rate of burning. Inasmuch as the engine can be timed late to eliminate detonation, it is now possible to use a fast burning fuel mixture with no lead or combustion deterrents. Since the combustion is taking place in a chamber which is large enough to contain the entire expanded charge at a satisfactory pressure, there is no reason to delay combustion thereby permitting the use of a fuel in a volatized condition as well as a lead free and pollutant free fuel.

The fact that the engine provides fast burning without excessive pressures and detonation makes it possible to use a high air-to-fuel ratio as well as volatized fuel in order to get complete combustion to eliminate carbon monoxide and unburned hydrocarbons. Additionally, since the pressure and temperature in the combustion chamber is limited, the excessive high temperatures which produce oxides of nitrogen are avoided. The efficiency of the engine is also improved because of the advantage that the completely expanded gas at the top of the expansion stroke is utilized throughout the full length of the expansion stroke instead of retarding part of the combustion until the piston has moved to a position of partial expansion. In summary, the shape of the cam permits a maximum compression at top center to provide ignition temperatures. The piston is then permitted to slightly move towards expansion to a position of dwell at which the combustion chamber is large enough to contain the total consumed charge without exceeding a desired maximum pressure. The position of dwell is maintained until combustion is substantially complete before expansion movement of the piston is continued to complete expansion and exhaust. The exact timing of the ignition is controlled by the point at which the fuel is injected similar to a conventional diesel, but the ignition is timed late enough to allow the piston to reach the dwell position ahead of the combustion pressure build-up.

Assuming that combustion has occurred and that the piston has moved past the dwell area, the piston begins to move outwardly towards a position of expansion with respect to the cylinder so that the roller moves from the dwell portion of the cam plate onto the cam surface indicated by **222** with the expansion movement of the piston causing the rotor **136** and shaft **26** to be rotated. The curvature or sweep of the cam surface at **222** is such that the piston is allowed to move towards a position of expansion at a comparatively rapid rate after the dwell and after substantially complete combustion. The piston continues to expand due to the combustion forces on the head thereof until the head of the piston has moved past the exhaust openings **182** in the cylinder to permit the exhaust gases to be exhausted outwardly therethrough. Such exhausting occurs with regard to the cam plate at approximately **230.**

As previously stated, air under pressure is constantly being supplied to the notches **46** and **48** to permit air to be forced into the cylinders when the notches **46** and **48** communicate therewith through the ports **237** during the rotation of the rotor and cylinders. The air under pressure is forced into the cylinder which aids in scavenging the exhaust gases from the cylinders when the top of the piston has been moved below the openings **182.** The dwell of the piston at the bottom of the stroke (**230**) is easily modified by the shape of the cam, and the time involved is at least 50% or more of the total time of a conventional 4-stroke cycle. During that entire time, ambient air from the air blower is being forced through the cylinder and out the exhaust holes **182** in the side of the cylinder. The air not only scavenges the exhaust from the cylinder but aids in cooling the engine as well as recharging the cylinder for the next cycle. The volume of air passing outwardly through the openings **182** is not nearly as much as is needed to cool a cylinder from its outside surface. However, the inside surface temperature of the cylinder (300° to 500°) is far higher than the outside surface of the cylinder, and therefore the heat transfer rate between the inside cylinder surface and the air is much faster than would be possible when cooling the cylinder from the outside. Therefore, it has been found that a lesser amount of air on the inside of the cylinder is quite adequate because of the greater temperature difference and its faster cooling rate. Furthermore, since the pistons are controlled by the shape of a cam, it is a simple matter to allow the compression stroke, and especially the expansion stroke, to be condensed into a shorter period of time so that less heat is lost into the walls of the cylinder therefore requiring less cooling. Further, as the compression and expansion time is shortened, more time is allowed for the cooling cycle.

The forcing of the air into the cylinders also achieves an additional result. In conventional rotary engines, centrifugal force is employed to maintain the pistons out against the cam. Centrifugal force alone is not enough to overcome the vacuum effect when the pistons are required to suck in their charge of air in the conventional manner. In this engine, the blower or air pump is used to charge the cylinders which forces the air into the cylinders under pressure so that rather than a vacuum which retards the outward movement of the pistons, a pressure is provided which adds to the centrifugal force to move the pistons out against the cam. The air pressure against the pistons insures that the rollers on the pistons will follow the cam as well as provide a supercharging effect.

The air fed to the interior of the cylinder is then compressed as the roller on the piston approaches **224** on the cam surface. The piston is moved inwardly into the cylinder thereby closing the openings **182** so that the air in the cylinder can be compressed. The air is compressed and the fuel is sprayed into the cylinder when the jets **118** and **120** register with ports **237** to cause the ignition thereof as previously described.

An extremely important feature of this invention is the fact that vibration problems are greatly reduced. In a conventional crank engine, the reciprocating pistons and other reciprocating parts all tend to cause vibration. The vibration problem is substantially reduced

due to the fact that two diametrically opposing pistons are moving in opposite directions at all times. Because there are two lobes on the cam, a complete cycle of operation is obtained twice in every revolution in each of the pistons. Furthermore, the mounting of the pistons diametrically opposite each other, and firing them together, causes one to neutralize the other in a way that a couple-type force is applied to the rotor so that the load on the bearings of the rotor is minimized. Additionally, the firing force against the piston is taken directly by, and in line with, the cam so that all of the forces of explosions are concentrated in the cam, which is the one member most capable of withstanding such forces. With reference to FIG. **6**, it is seen that the dimension A which defines the effective width of the sealing area of rotor **136** on core member **34** is less than the dimension B which is the effective diameter of the pressure area on the bottom of the cylinders. Thus, the combustion forces in the cylinder cause the cylinders to urge the rotor member **138** inwardly into sealing engagement with the stationary core member **34** to provide a positive seal.

Thus, it can be seen that the rotary internal combustion of this invention involves an unique method to cool the cylinders thereof by introducing cool air to the interior thereof over a substantial portion of the engine cycle, thus resulting in a much improved engine of greater efficiency. It can also be seen that a novel means has been provided for achieving maximum combustion efficiency by providing a slight dwell area beyond top dead center on the cam lobes. The air fed through the cylinders not only cools the cylinder but aids in forcing the pistons out against the cam plate. Thus it can be seen that a rotary combustion engine has been provided which accomplishes at least all of its stated objectives.

I claim:

1. A rotary internal combustion engine comprising,

an engine frame means including a rotatable power output means,

a rotor means in said frame means connected to said power output means, said rotor means having a central opening,

a plurality of cylinders mounted on said rotor means,

a piston movably mounted in each of said cylinders and being movable between compression and expansion positions with respect thereto,

a cam plate means on said frame means extending around said rotor means and having a cam surface,

a roller means mounted on each of said pistons which engages and rolls upon said cam surface to cause said piston to be moved in said cylinder as said rotor means is rotated with respect to said cam plate,

a core means having at least a portion thereof received by said central opening of said rotor means, said rotor means rotatably embracing said core means;

said core means having an air passageway means extending therethrough for supplying air to said cylinders,

said core means having a fuel supply means extending therethrough for supplying fuel to said cylinders,

said rotor means defining a sealing area with respect to said core means, said cylinders having a compression area in said rotor means adjacent said sealing area; the compression area in said cylinders being greater than the sealing area between said rotor means and said core means; said rotor means being flexible so that the combustion forces in said compression area cause the cylinders to urge the rotor means inwardly into sealing engagement with said core means.

2. The structure of claim **1** wherein said engine comprises means for introducing lubricant into said central opening between said core means and said rotor upon said rotor means being moved outwardly from said core means.

3. The method of providing a seal between a stationary core means and the rotor means of a rotary internal combustion engine, comprising,

extending at least a portion of the core means through a central opening formed in a rotor means so that the rotor means rotatably embraces the core means,

combusting the fuel in the cylinders of the engine so that the combustion forces in the cylinders urge the rotor means towards the core means to create a seal therebetween.

4. The method of claim **3** wherein said rotor means is alternately moved towards and away from the core means so that lubricant may be introduced between the core means and the rotor means when the rotor means is moved away from the core means.

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