

FLUID MACHINE HAVING A LABYRINTH SEAL**Field of the invention**

5 The present invention relates to a fluid machine, for example a centrifugal compressor, including a labyrinth seal defined between a rotating member and a stationary member of the fluid machine, and configured to minimize fluid flow between a high pressure section and a low pressure section formed within a housing of the fluid machine.

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Background of the invention

US5244216 discloses a fluid machine including:

- a housing,
- 15 - a rotational shaft located within the housing,
- a stationary member connected to the housing,
- a rotary member connected to the rotational shaft, and
- a labyrinth seal configured to minimize or control fluid flow across the labyrinth seal, the labyrinth seal including:
 - 20 - a succession of stationary steps formed on the stationary member, each stationary step including a radial wall portion extending substantially parallelly to a longitudinal axis of the rotational shaft and an axial wall portion extending substantially perpendicularly to the longitudinal axis of the rotational shaft,
 - a succession of rotary steps formed on the rotary member, each
 - 25 rotary step including a radial wall portion extending substantially parallelly to the longitudinal axis of the rotational shaft and an axial wall portion extending substantially perpendicularly to the longitudinal axis of the rotational shaft,
 - a plurality of stationary recesses, each stationary recess being formed in the radial wall portion of a respective stationary step, each stationary step
 - 30 defining a stationary projection delimited by the stationary recess formed on said stationary step and by the axial wall portion of said stationary step, and
 - a plurality of rotary recesses, each rotary recess being formed in the radial wall portion of a respective rotary step and adjacent to the axial wall portion of an adjacent rotary step located upstream said respective rotary step, each
 - 35 rotary step defining a rotary projection delimited by the rotary recess formed on said rotary step and by the axial wall portion of said rotary step.

Such a configuration of the labyrinth seal induces "obstacles" and restrictions to the leakage flow, and thus reduces fluid leakage through the labyrinth seal.

However such a labyrinth seal has flow passages and cavities of complicated shapes, which reduces the number of stationary and rotary steps available on a given axial sealing length, and which are complicated to manufacture. Further, such a labyrinth seal requires very accurate mounting and axial orientation between the rotating and stationary members. Furthermore, the configuration of such a labyrinth seal would be too fragile to be transferred to small and compact fluid machines.

Summary of the invention

It is an object of the present invention to provide a fluid machine having an improved labyrinth seal which can overcome the drawbacks encountered with conventional labyrinth seals.

Another object of the present invention is to provide a robust, efficient and easy to manufacture labyrinth seal having a simple design which allows large tolerances on axial dimensions of the stationary and rotary members and large relative axial displacement between the stationary and rotary members. This is especially important in very small and compact fluid machines, where the total axial length of the labyrinth seal is a few millimeters and hence the axial length of each step is a few hundred microns.

According to the invention such a fluid machine includes:

- a housing,
- a rotational shaft located within the housing,
- a stationary member connected to the housing,
- a rotary member connected to the rotational shaft,
- a labyrinth seal configured to minimize or control fluid flow across the labyrinth seal, the labyrinth seal including:

- a succession of stationary steps formed on the stationary member, each stationary step of said succession of stationary steps including a radial wall portion extending substantially parallelly to a longitudinal axis of the rotational shaft and an axial wall portion extending substantially perpendicularly to the longitudinal axis of the rotational shaft,

- a succession of rotary steps formed on the rotary member, each rotary step of said succession of rotary steps including a radial wall portion extending substantially parallelly to the longitudinal axis of the rotational shaft and an axial wall portion extending substantially perpendicularly to the longitudinal axis of the rotational shaft,

- a plurality of stationary recesses, each stationary recess of said plurality of stationary recesses being formed in the radial wall portion of a respective stationary step and adjacent to the axial wall portion of an adjacent stationary step located downstream said respective stationary step, each stationary step being configured to define a stationary projection delimited by the stationary recess formed on said stationary step and by the axial wall portion of said stationary step,

- a plurality of rotary recesses, each rotary recess of said plurality of rotary recesses being formed in the radial wall portion of a respective rotary step and adjacent to the axial wall portion of an adjacent rotary step located upstream said respective rotary step, each rotary step being configured to define a rotary projection delimited by the rotary recess formed on said rotary step and by the axial wall portion of said rotary step, and

wherein the axial width of each of the stationary recesses and of the rotary recesses substantially equals the axial width of each of the stationary projections and of the rotary projections.

Such a configuration, and particularly such a sizing of the stationary and rotary recesses and of the stationary and rotary projections, improves the efficiency and the robustness of the labyrinth seal. Further such a configuration of the stationary and rotary recesses and of the stationary and rotary projections reduces the manufacturing constraints for manufacturing the stationary and rotary members, and thus reduces the manufacturing cost of the fluid machine.

Furthermore, such a configuration of the labyrinth seal eases the assembly of the stationary and rotary members, and reduces the axial motion sensitivity of the rotary member during assembly and during operation of the fluid machine.

In the present patent application, the wordings “downstream” and “upstream” and similar wordings are considered with respect to the direction of the fluid flow through the labyrinth seal.

The fluid machine may also include one or more of the following features, taken alone or in combination.

According to an embodiment of the invention, the labyrinth seal is configured to minimize or control fluid flow between a high pressure section and a low pressure section formed within the housing.

According to an embodiment of the invention, the radial wall portions
5 and the axial wall portions of the stationary steps and of the rotary steps are substantially planar.

According to an embodiment of the invention, the radial wall portions of the stationary steps and of the rotary steps extend parallelly to the longitudinal axis of the rotational shaft. According to another embodiment of the invention, the radial
10 wall portions of the stationary steps and of the rotary steps extend at an angle between about -5 degrees and about 5 degrees with respect to the longitudinal axis of the rotational shaft.

According to an embodiment of the invention, the axial wall portions of the stationary steps and of the rotary steps extend perpendicularly to the longitudinal
15 axis of the rotational shaft. According to another embodiment of the invention, the axial wall portions of the stationary steps and of the rotary steps extend at an angle between about 85 degrees and about 95 degrees with respect to the longitudinal axis of the rotational shaft.

According to an embodiment of the invention, the axial width of each of
20 the stationary recesses and of the rotary recesses equals the axial width of each of the stationary projections and of the rotary projections. According to another embodiment of the invention, the ratio between the axial width of each of the stationary recesses and of the rotary recesses and the axial width of each of the stationary projections and of the rotary projections is between 0.8 and 1.2, or
25 between 0.9 and 1.1.

According to another embodiment of the invention, the ratio between the average of the axial widths of the stationary recesses and of the rotary recesses and the average of the axial widths of the stationary projections and of the rotary
projections is between 0.8 and 1.2, or between 0.9 and 1.1.

According to an embodiment of the invention, the stationary steps are axially offset from each other and successively arranged along an inner radial surface of the stationary member, and the rotary steps are axially offset from each other and successively arranged along an outer radial surface of the rotary member.

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According to an embodiment of the invention, the inner radial surface of the stationary member and the outer radial surface of the rotary member are opposite to each other.

According to an embodiment of the invention, the stationary steps and
5 the rotary steps are annular.

According to an embodiment of the invention, the stationary recesses and the rotary recesses are annular.

According to an embodiment of the invention, the stationary projections and the rotary projections are annular.

10 According to an embodiment of the invention, the inner radial surface of the stationary member and the outer radial surface of the rotary member are annular.

According to an embodiment of the invention, each stationary projection extends substantially perpendicularly to the longitudinal axis of the rotational shaft, and each rotary projection extends substantially perpendicularly to the longitudinal
15 axis of the rotational shaft.

According to an embodiment of the invention, each stationary projection at least partially faces a respective rotary projection.

According to an embodiment of the invention, each stationary projection extends radially inward towards the respective rotary projection, and each rotary
20 projection extends radially outward towards the respective stationary projection.

According to an embodiment of the invention, each stationary projection and the respective rotary projection delimit a flow passage extending substantially parallelly to the longitudinal axis of the rotational shaft.

According to an embodiment of the invention, each stationary projection
25 includes an upstream edge and a downstream edge and each rotary projection includes an upstream edge and a downstream edge, the upstream edge of each stationary projection being axially offset from the upstream edge of the respective rotary projection by an axial distance.

According to an embodiment of the invention, the upstream edge of each
30 stationary projection is upstreamly axially offset from the upstream edge of the respective rotary projection.

According to an embodiment of the invention, the upstream and downstream edges of each stationary projection are sharp, and the upstream and downstream edges of each rotary projection are sharp.

According to an embodiment of the invention, the axial distance is larger than the maximum allowed relative axial movement between the rotary member and the stationary member during operation of the fluid machine.

According to an embodiment of the invention, each stationary recess at
5 least partially faces a respective rotary recess.

According to an embodiment of the invention, the fluid machine further includes a succession of cavities each having a flow inlet and a flow outlet, each cavity being partially delimited by a stationary recess and a rotary recess, the flow inlet of each cavity being positioned at a location between the respective stationary recess
10 and the axial wall portion of an upstream rotary step.

According to an embodiment of the invention, the flow inlet of each cavity is facing the axial wall portion of the downstream stationary step.

According to an embodiment of the invention, each cavity is further delimited by the axial wall portion of an adjacent stationary step located downstream
15 said cavity and by the axial wall portion of an adjacent rotary step located upstream said cavity.

According to an embodiment of the invention, each cavity is annular.

According to an embodiment of the invention, the labyrinth seal is configured such that the pressure in a downstream cavity is lower than the pressure in
20 an upstream cavity.

According to an embodiment of the invention, each stationary recess has a rounded or curved concave bottom and each rotary recess has a rounded or curved concave bottom.

According to an embodiment of the invention, the stationary member
25 has a disc shape.

According to an embodiment of the invention, the stationary member is made in one piece and is assembled in an axial direction in relation to the rotary member.

According to an embodiment of the invention, the stationary member
30 made in two separated pieces and is assembled in a radial direction around the rotary member.

According to an embodiment of the invention, the labyrinth seal includes a flow intake located upstream the cavities and configured to supply the cavities with a fluid flow.

According to an embodiment of the invention, the fluid machine is a
35 centrifugal compressor, a turbine or a pump.

According to an embodiment of the invention, the rotary member is integrally formed with the rotational shaft.

According to an embodiment of the invention, the stationary and rotary steps have substantially the same shape, and the stationary and rotary recesses have substantially the same shape. This configuration allows notably to manufacture the both sides of the labyrinth seal with the same tool.

These and other advantages will become apparent upon reading the following description in view of the drawing attached hereto representing, as non-limiting example, an embodiment of a fluid machine according to the invention.

Brief description of the drawings

The following detailed description of one embodiment of the invention is better understood when read in conjunction with the appended drawings being understood, however, that the invention is not limited to the specific embodiment disclosed.

Figure 1 and 2 are partial section views of a fluid machine according to a first embodiment of the invention.

Figure 3 is a front view of a stationary member of the fluid machine of figure 1.

Figure 4 is an enlarged view of a detail of figure 1.

Figure 5 is an enlarged view of a detail of figure 4.

Figure 6 is a partial section view of a fluid machine according to a second embodiment of the invention.

Figure 7 is a front view of a stationary member of the fluid machine of figure 6.

Detailed description of the invention

In the description which follows, the same elements are designated with the same references in the different embodiments.

Figure 1 to 5 discloses a fluid machine 2, for example a centrifugal compressor, a turbine or a pump, including a housing 3 and a rotational shaft 4 rotatably mounted within the housing 3 and rotatably supported by radial bearings and axial bearing (not shown on the figures). The axial bearings are configured to

allow a very limited axial movement of the rotational shaft during operation of the fluid machine 2.

The fluid machine 2 further includes a rotary member 5 fixed to the rotational shaft 4, and a stationary member 6 connected to the housing 3 and
5 surrounding the rotary member 5. The housing 3 may for example include a fixed support (not shown on the figures) on which is fixed the stationary member 6.

As better shown on figure 3, the stationary member 6 has a disc shape, and includes an inner peripheral surface 7 and an outer peripheral surface 8. According to the embodiment shown on figures 1 to 5, the stationary member 6 is
10 made in one piece, and it is assembled in an axial direction between two parts 5a and 5b of the rotary member 5.

The fluid machine 2 furthermore includes a stepped labyrinth seal 9 configured to minimize or control fluid flow between a high pressure section 27 and a low pressure section 29 formed within the housing 3.

The labyrinth seal 9 includes a succession of annular stationary steps 11
15 axially offset from each other and formed on the inner peripheral surface 7 of the stationary member 6. Each stationary step 11 includes an annular radial wall portion 12 extending parallelly to a longitudinal axis A of the rotational shaft 4 and an annular axial wall portion 13 extending perpendicularly to the longitudinal axis A of the rotational shaft 4. The radial wall portions 12 and the axial wall portions 13 of the
20 stationary steps 11 are advantageously substantially planar.

The labyrinth seal 9 includes a succession of annular rotary steps 14 axially offset from each other and formed on an outer peripheral surface 15 of the rotary member 5 which is opposite to and adjacent to the inner peripheral surface 7
25 of the stationary member 6. Each rotary step 14 includes an annular radial wall portion 16 extending parallelly to the longitudinal axis A of the rotational shaft 4 and an annular axial wall portion 17 extending perpendicularly to the longitudinal axis A of the rotational shaft 4. The radial wall portions 16 and the axial wall portions 17 of the rotary steps 14 are advantageously substantially planar.

The labyrinth seal 9 also includes a plurality of annular stationary recesses 18 formed on the stationary member 6. Each stationary recess 18 is formed in the radial wall portion 12 of a respective stationary step 11 and adjacent to the axial wall portion 13 of an adjacent stationary step 11 located downstream said respective stationary step 11. Each stationary recess 18 has advantageously a rounded or curved
35 concave bottom.

Each stationary step 11 thus defines an annular stationary tooth or projection 19 delimited by the stationary recess 18 formed on said stationary step 11 and by the axial wall portion 13 of said stationary step 11. Each stationary projection 19 extends perpendicularly to the longitudinal axis A of the rotational shaft 4, and includes an upstream edge and a downstream edge.

The labyrinth seal 9 also includes a plurality of annular rotary recesses 21 formed on the rotary member 5. Each rotary recess 21 is formed in the radial wall portion 16 of a respective rotary step 14 and adjacent to the axial wall portion 17 of an adjacent rotary step 14 located upstream said respective rotary step 14. Each rotary recess 21 has advantageously a rounded or curved concave bottom. As better shown on figure 5, each rotary recess 21 at least partially faces a respective stationary recess 18.

Each rotary step 14 thus defines an annular rotary tooth or projection 22 delimited by the rotary recess 21 formed on said rotary step 14 and by the axial wall portion 17 of said rotary step 14. Each rotary projection 22 extends perpendicularly to the longitudinal axis A of the rotational shaft A, and includes an upstream edge and a downstream edge.

Each rotary projection 22 at least partially faces a respective stationary projection 19. Particularly, each stationary projection 19 extends radially inward towards the respective rotary projection 22, and each rotary projection 22 extends radially outward towards the respective stationary projection 19. Further each stationary projection 19 and the respective rotary projection 22 delimit an annular flow passage 23 extending parallelly to the longitudinal axis A of the rotational shaft 4.

As better shown on figure 5, the upstream edge of each stationary projection 19 is upstreamly axially offset from the upstream edge of the respective rotary projection 22 by an axial distance D. Advantageously, the axial distance D is larger than the maximum allowed axial movement of the rotary member 5 during operation of the fluid machine.

Further, as shown on figure 5, the axial width W_r of each of the stationary recesses 18 and of the rotary recesses 21 substantially equals the axial width W_p of each of the stationary projections 19 and of the rotary projections 22.

The fluid machine 2 further includes a succession of annular cavities 24 each having a flow inlet 25 and a flow outlet 26. The flow outlet 26 and the flow inlet 25 of two adjacent cavities are fluidly connected by a respective flow passage 23.

Each cavity 24 is delimited by a stationary recess 18, a rotary recess 21, the axial wall portion 13 of an adjacent stationary step 11 located downstream said

cavity 24 and by the axial wall portion 17 of an adjacent rotary step 14 located upstream said cavity 24. The flow inlet 25 of each cavity 24 is positioned at a location between the respective stationary recess 18 and the respective axial wall portion 17, and is facing the respective axial wall portion 13.

5 The fluid machine 2 further includes a high pressure chamber 27 defined by the rotary and stationary members 5, 6, and located upstream the cavities 24. The labyrinth seal 9 includes a flow intake 28 fluidly connected to the high pressure chamber 27, and configured to supply the cavities 24 with a fluid flow coming from the high pressure chamber 27.

10 The fluid machine 2 also includes a low pressure chamber 29 defined by the rotary and stationary members 5, 6, and located downstream the cavities 24. The labyrinth seal 9 includes a flow output 31 fluidly connected to the low pressure chamber 29.

In use, a fluid flow enters the first cavity 24, which is adjacent to the high
15 pressure chamber 27, via the flow intake 28 and the flow inlet 25 of said first cavity 24. As the flow inlet 25 is facing the axial wall portion 13 of the downstream stationary step 11, the fluid flow impacts said axial wall portion 13. Such an impact generates a first recirculation flow F1 within an upper portion of the first cavity 24, a second recirculation flow F2 within a lower portion of the first cavity 24, and a leakage
20 flow F3 going down towards the flow outlet 26 of the first cavity 24 in a direction substantially parallel to axial wall portion 13. Due to the directional change at the flow outlet 26, a third recirculation flow F4 is created at the offsetted edges of stationary projection 19 and rotary projection 22. The third recirculation flow F4 is reducing the effective flow area for the leakage flow F3 leaving the cavity 24 at flow outlet 26. After
25 having passed flow passage 23, the leakage flow F3 then enters the second cavity 24 where a first recirculation flow, a second and a third recirculation flow and a leakage flow are also generated, and where the leakage flow enters the third cavity 24, and so on.

The generation of a first recirculation flow F1, a second recirculation flow
30 F2, a third recirculation flow F3 and a leakage flow F4 in the successive cavities 24 reduces largely the fluid leakage through the labyrinth seal 9, and allows obtaining a low leakage rate through the labyrinth seal 9.

Figures 6 and 7 disclose a fluid machine 2 according to a second
embodiment of the invention which differs from the embodiment shown on figures 1
35 to 5 essentially in that the stationary member 6 is made in two separated pieces 6a,

6b, and that is assembled in a radial direction around the rotary member 5, which is made as a unitary part.

Of course, the invention is not restricted to the embodiments described above by way of non-limiting examples, but on the contrary it encompasses all
5 embodiments thereof.

CLAIMS

1. Fluid machine (2) including:
- a housing (3),
 - 5 - a rotational shaft (4) located within the housing (3),
 - a stationary member (6) connected to the housing (3),
 - a rotary member (5) connected to the rotational shaft (4),
 - a labyrinth seal (9) configured to minimize or control fluid flow across the labyrinth seal (9), the labyrinth seal (9) including:
 - 10 - a succession of stationary steps (11) formed on the stationary member (6), each stationary step (11) of said succession of stationary steps (11) including a radial wall portion (12) extending substantially parallel to a longitudinal axis (A) of the rotational shaft (4) and an axial wall portion (13) extending substantially perpendicularly to the longitudinal axis (A) of the rotational shaft (4),
 - 15 - a succession of rotary steps (14) formed on the rotary member (5), each rotary step (14) of said succession of rotary steps (14) including a radial wall portion (16) extending substantially parallel to the longitudinal axis (A) of the rotational shaft (4) and an axial wall portion (17) extending substantially perpendicularly to the longitudinal axis (A) of the rotational shaft (4),
 - 20 - a plurality of stationary recesses (18), each stationary recess (18) of said plurality of stationary recesses (18) being formed in the radial wall portion (12) of a respective stationary step (11) and adjacent to the axial wall portion (13) of an adjacent stationary step (11) located downstream said respective stationary step (11), each stationary step (11) being configured to define a stationary projection (19)
 - 25 delimited by the stationary recess (18) formed on said stationary step (11) and by the axial wall portion (13) of said stationary step (11), and
 - a plurality of rotary recesses (21), each rotary recess (21) of said plurality of rotary recesses (21) being formed in the radial wall portion (16) of a
 - 30 respective rotary step (14) and adjacent to the axial wall portion (17) of an adjacent rotary step (14) located upstream said respective rotary step (14), each rotary step (14) being configured to define a rotary projection (22) delimited by the rotary recess (21) formed on said rotary step (14) and by the axial wall portion (17) of said rotary step (14),
 - wherein the axial width (Wr) of each of the stationary recesses (18) and
 - 35 of the rotary recesses (21) substantially equals the axial width (Wp) of each of the stationary projections (19) and of the rotary projections (22).

2. The fluid machine (2) according to claim 1, wherein the stationary steps (11) are axially offset from each other and successively arranged along an inner radial surface (7) of the stationary member (6), and the rotary steps (14) are axially offset from each other and successively arranged along an outer radial surface (15) of the rotary member (5).

3. The fluid machine (2) according to claim 1 or 2, wherein each stationary projection (19) extends substantially perpendicularly to the longitudinal axis (A) of the rotational shaft (4), and each rotary projection (22) extends substantially perpendicularly to the longitudinal axis (A) of the rotational shaft (4).

4. The fluid machine (2) according to any one of claim 1 to 3, wherein each stationary projection (19) at least partially faces a respective rotary projection (22).

5. The fluid machine (2) according to claim 4, wherein each stationary projection (19) and the respective rotary projection (22) delimit a flow passage (23) extending substantially parallelly to the longitudinal axis (A) of the rotational shaft (4).

6. The fluid machine (2) according to claim 4 or 5, wherein each stationary projection (19) includes an upstream edge and a downstream edge and each rotary projection (22) includes an upstream edge and a downstream edge, the upstream edge of each stationary projection (19) being axially offset from the upstream edge of the respective rotary projection (22) by an axial distance (D).

7. The fluid machine (2) according to claim 6, wherein the axial distance (D) is larger than the maximum allowed relative axial movement between the rotary member (5) and the stationary member (6) during operation of the fluid machine (2).

8. The fluid machine (2) according to claim 6 or 7, wherein the upstream and downstream edges of each stationary projection are sharp, and the upstream and downstream edges of each rotary projection are sharp.

9. The fluid machine (2) according to any one of claims 1 to 8, wherein each stationary recess (18) at least partially faces a respective rotary recess (21).

10. The fluid machine (2) according to any one of claims 1 to 9, further including a succession of cavities (24) each having a flow inlet (25) and a flow outlet (26), each cavity (24) being partially delimited by a stationary recess (18) and a rotary recess (21), the flow inlet (25) of each cavity (24) being positioned at a location between the respective stationary recess (18) and the axial wall portion (17) of an upstream rotary step (14).

11. The fluid machine (2) according to claim 10, wherein the flow inlet (25) of each cavity (24) is facing the axial wall portion (13) of the downstream stationary step (18).

12. The fluid machine (2) according to any one of claims 1 to 11, wherein the fluid machine (2) is a centrifugal compressor, a turbine or a pump.

13. The fluid machine (2) according to any one of claims 1 to 12, wherein the rotary member (5) is integrally formed with the rotational shaft (4).

14. The fluid machine (2) according to any one of claims 1 to 13, wherein the stationary and rotary steps have substantially the same shape, and the stationary and rotary recesses have substantially the same shape.

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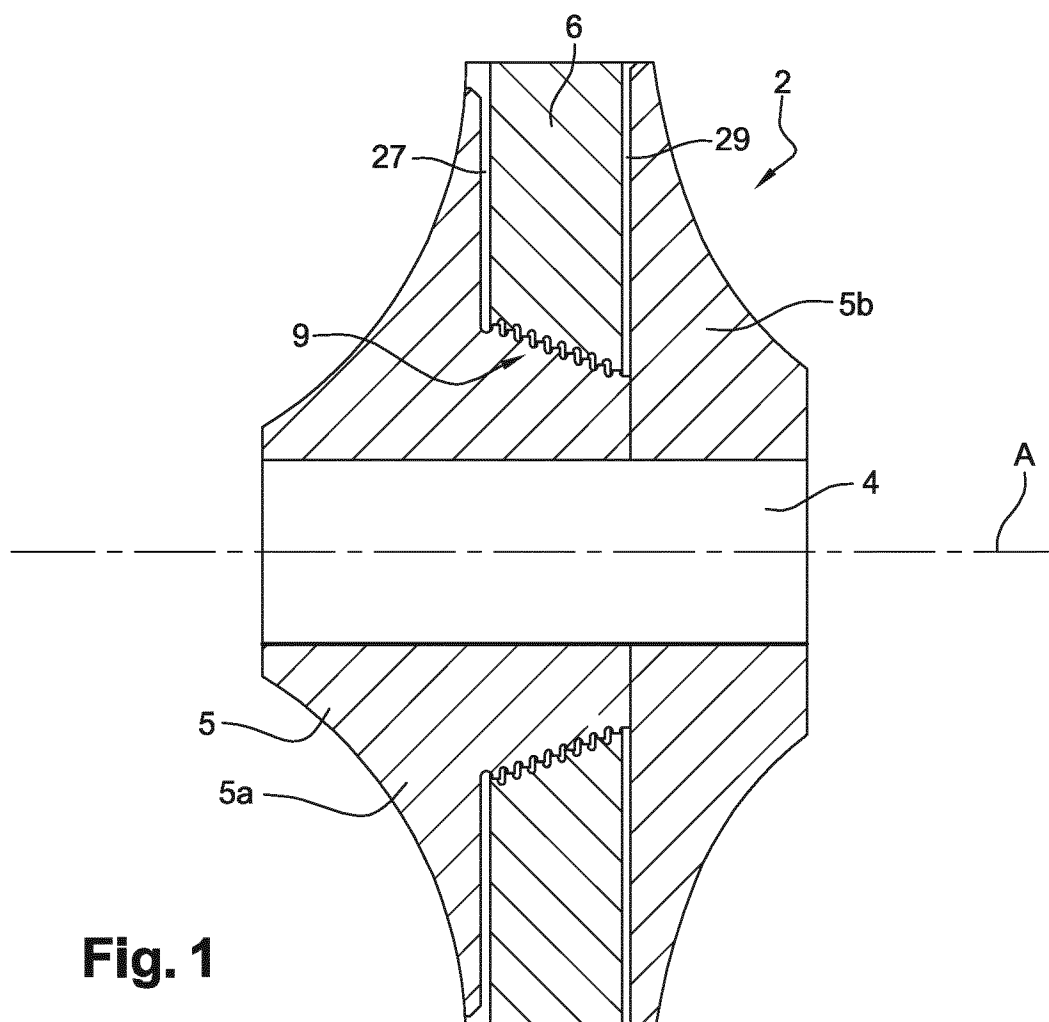


Fig. 1

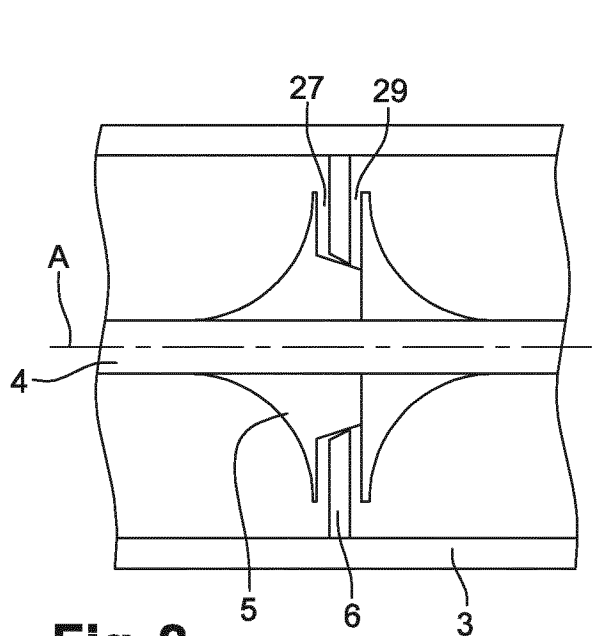


Fig. 2

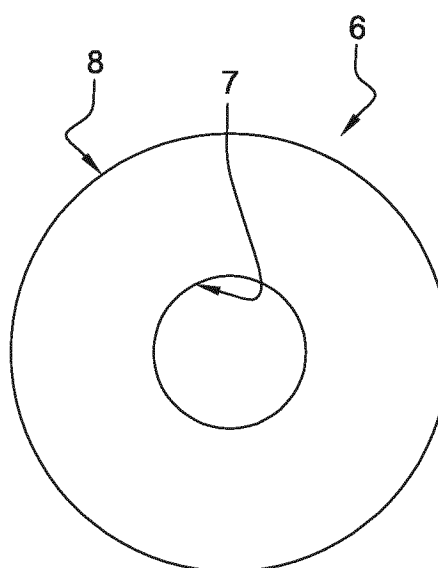
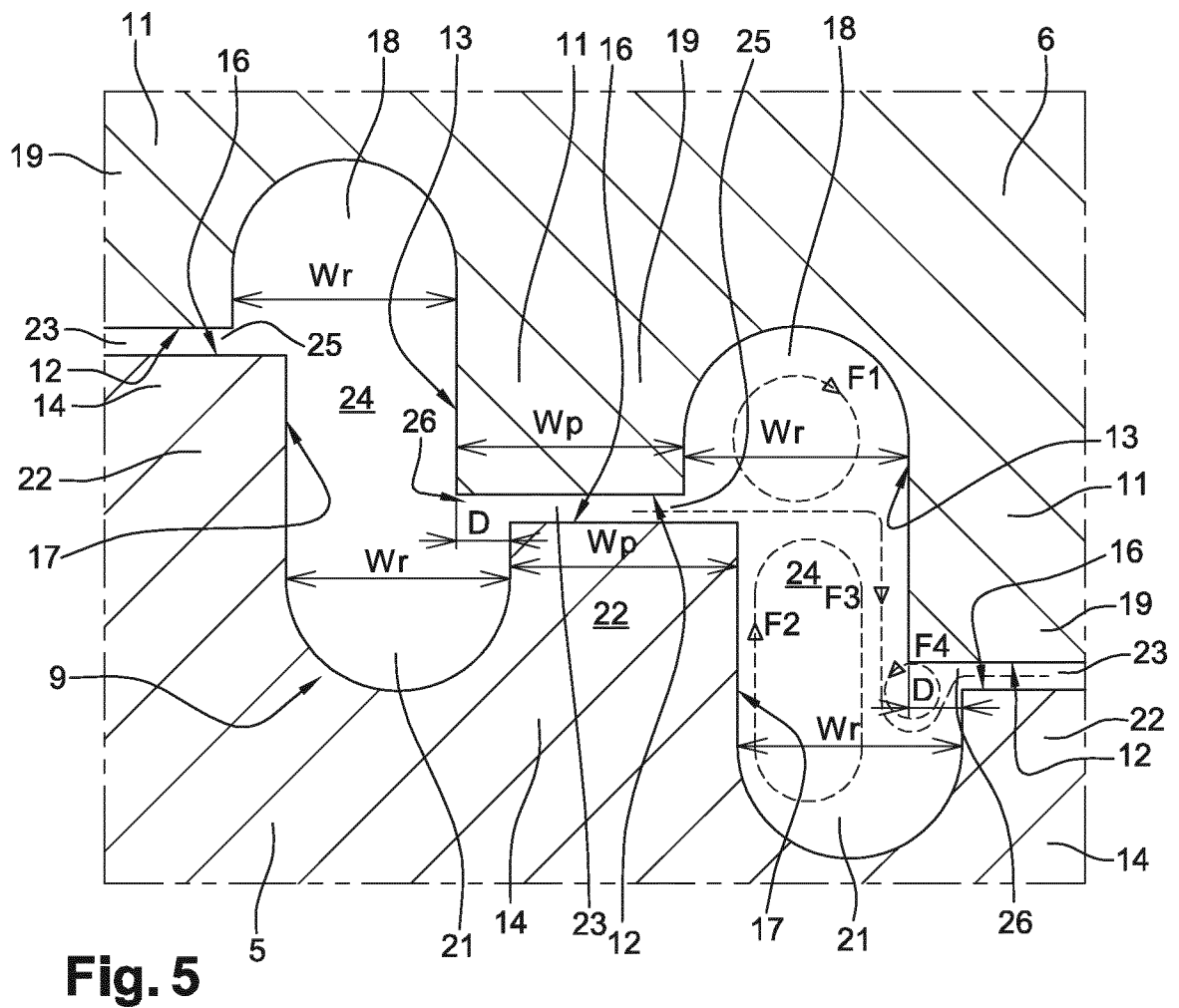
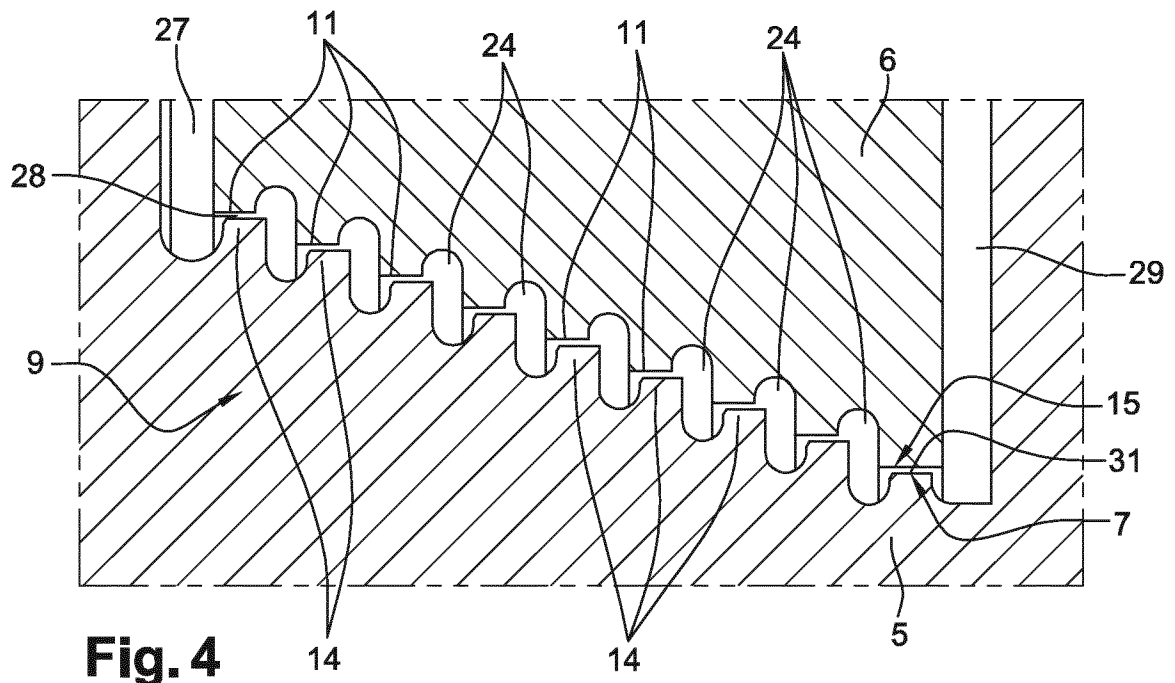


Fig. 3

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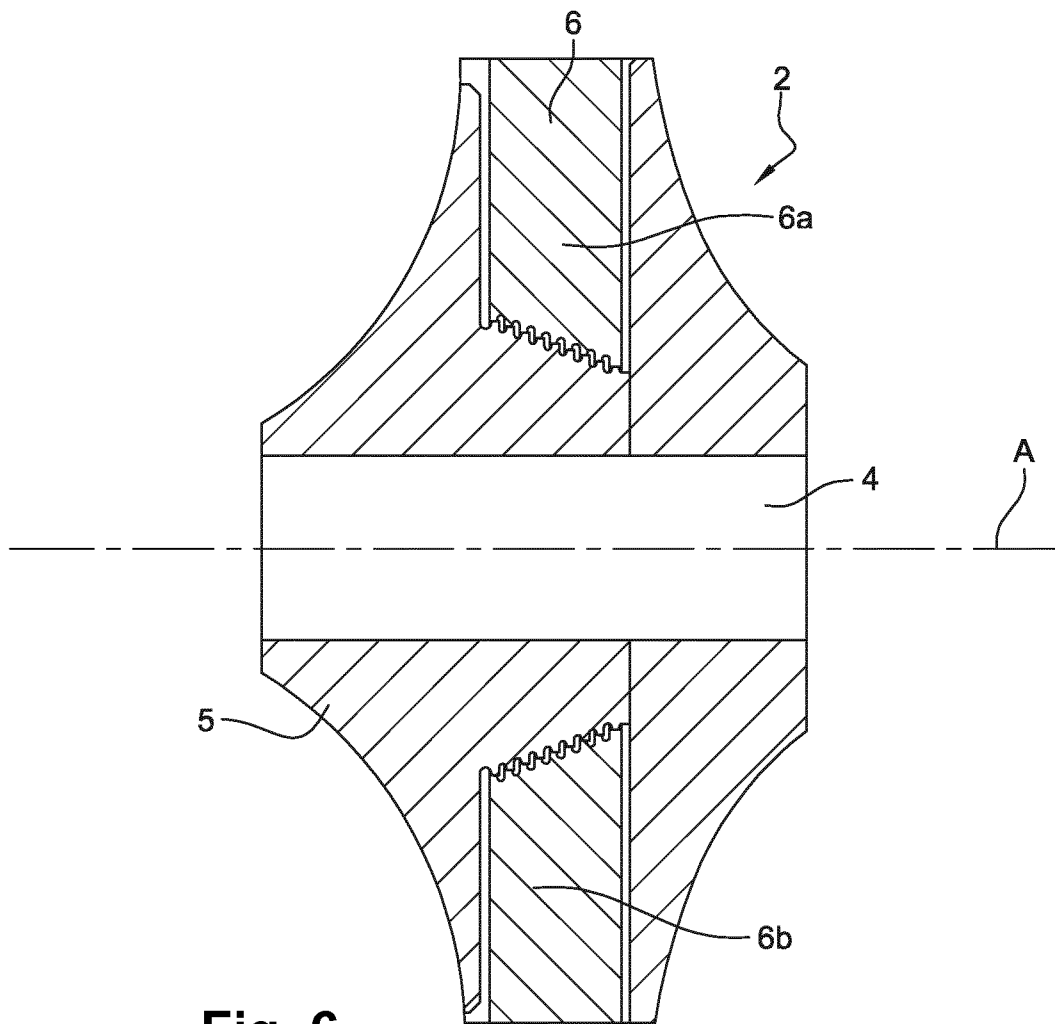


Fig. 6

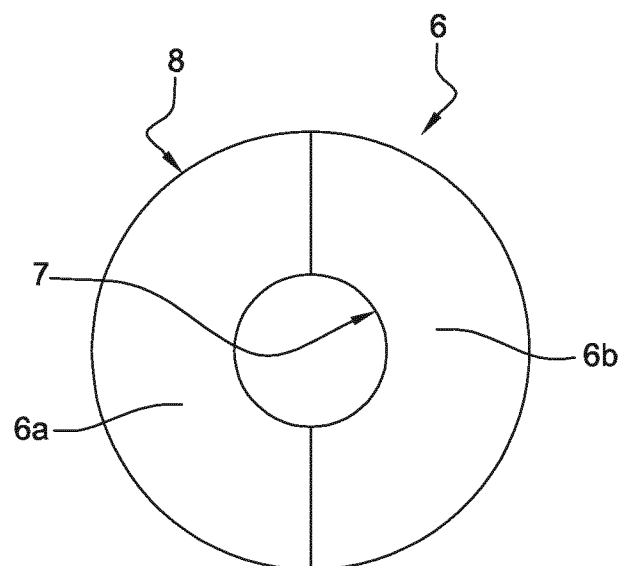


Fig. 7

INTERNATIONAL SEARCH REPORT

International application No

PCT/EP2016/061183

A. CLASSIFICATION OF SUBJECT MATTER
INV. F16J15/447
ADD.

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)
F16J

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)

EPO-Internal

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	WO 2012/097839 A1 (VOITH PATENT GMBH [DE]; STUMMER MANFRED [DE]) 26 July 2012 (2012-07-26)	1-12,14
Y	page 5, line 4 - page 6, line 21; figures 1-4b	13
Y	----- US 3 231 285 A (WELTNER WILTON W ET AL) 25 January 1966 (1966-01-25)	13
A	column 1, line 45 - column 2, line 24; figure 1	1
X	----- US 5 823 740 A (CYBULARZ JOSEPH M [US] ET AL) 20 October 1998 (1998-10-20) column 3, line 60 - column 6, line 8; figures 1B, 2D	1-5, 9-12,14
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Further documents are listed in the continuation of Box C.



See patent family annex.

* Special categories of cited documents :

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Date of the actual completion of the international search

18 July 2016

Date of mailing of the international search report

01/08/2016

Name and mailing address of the ISA/

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INTERNATIONAL SEARCH REPORT

International application No

PCT/EP2016/061183

C(Continuation). DOCUMENTS CONSIDERED TO BE RELEVANT		
Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
A	US 5 244 216 A (RHODE DAVID L [US]) 14 September 1993 (1993-09-14) cited in the application column 5, line 44 - line 51; figure 2 -----	1

INTERNATIONAL SEARCH REPORT

Information on patent family members

International application No

PCT/EP2016/061183

Patent document cited in search report	Publication date	Patent family member(s)	Publication date
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		US 5924844 A	20-07-1999

US 5244216 A	14-09-1993	NONE	
