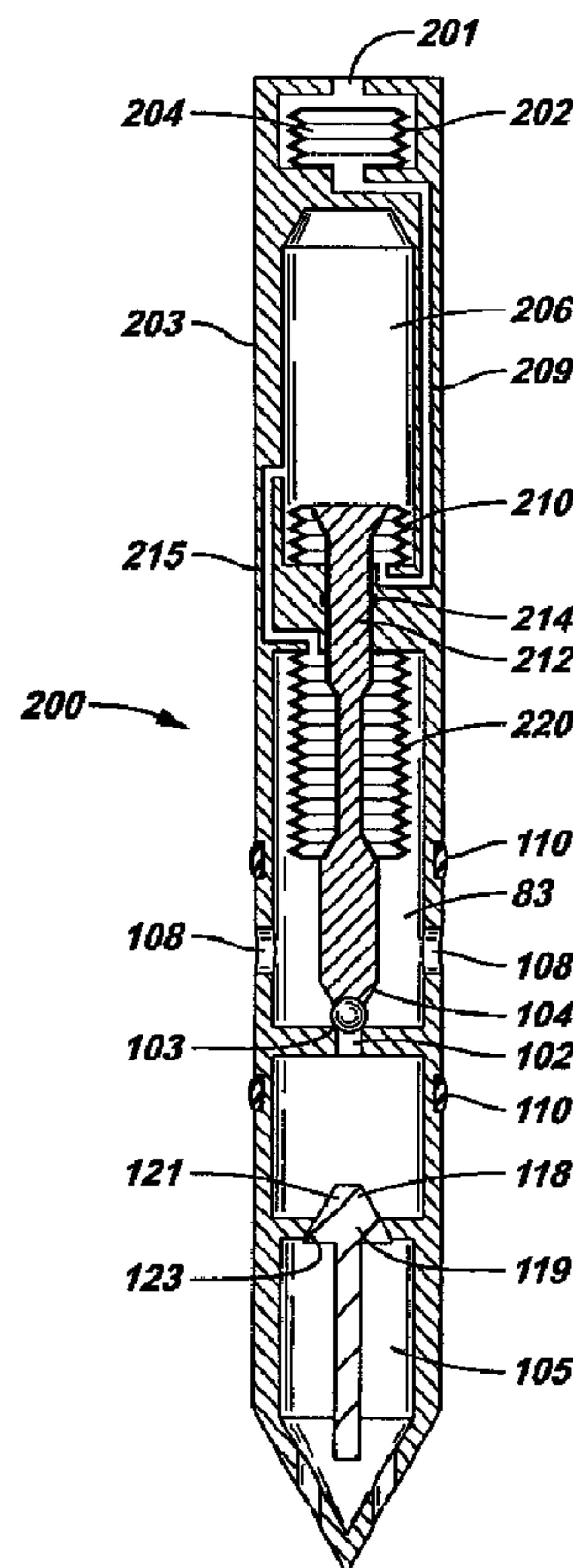




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(54) **Titre : SOUPAPE DE GAZ-LIFT**
 (54) **Title: GAS LIFT VALVE**



(57) **Abrégé/Abstract:**

A gas lift valve that is usable with a subterranean well includes a housing, a valve stem and at least one bellows. The housing has a port that is in communication with a first fluid, and the valve stem is responsive to the first fluid to establish a predefined threshold to open the valve. The bellow(s) form a seal between the valve stem and the housing. The bellow(s) are subject to a force that is exerted by the first fluid; and a second fluid contained in the bellow(s) opposes the force that is exerted by the first fluid.

GAS LIFT VALVE

ABSTRACT OF THE DISCLOSURE

A gas lift valve that is usable with a subterranean well includes a housing, a valve stem and at least one bellows. The housing has a port that is in communication with a first fluid, and the valve stem is responsive to the first fluid to establish a predefined
5 threshold to open the valve. The bellow(s) form a seal between the valve stem and the housing. The bellow(s) are subject to a force that is exerted by the first fluid; and a second fluid contained in the bellow(s) opposes the force that is exerted by the first fluid.

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GAS LIFT VALVE

RELATED APPLICATION

5 This application is a divisional of Canadian Patent Application
No. 2,461,485 filed March 19, 2004 and claims priority from therein.

BACKGROUND

The invention generally relates to a gas lift valve.

10 For purposes of communicating well fluid to a surface of a well, the well may
include a production tubing. More specifically, the production tubing typically extends
downhole into a wellbore of the well for purposes of communicating well fluid from one
or more subterranean formations through a central passageway of the production tubing to
the surface of the well. Due to its weight, the column of well fluid that is present in the
15 production tubing may suppress the rate at which the well fluid is produced from the
formation. More specifically, the column of well fluid inside the production tubing exerts
a hydrostatic pressure that increases with well depth. Thus, near a particular producing
formation, the hydrostatic pressure may be significant enough to substantially slow down
the rate at which the well fluid is produced from the formation.

20 For purposes of reducing the hydrostatic pressure and thus, enhancing the rate at
which fluid is produced, an artificial-lift technique may be employed. One such
technique involves injecting gas into the production tubing to displace some of the well
fluid in the tubing with lighter gas. The displacement of the well fluid with the lighter
gas reduces the hydrostatic pressure inside the production tubing and allows reservoir
fluids to enter the wellbore at a higher flow rate. The gas to be injected into the
production tubing typically is conveyed downhole via the annulus (the annular space
surrounding the production tubing) and enters the production tubing through one or more
gas lift valves.

As an example, Fig. 1 depicts a gas lift system 10 that includes a production
tubing 14 that extends into a wellbore. For purposes of gas injection, the system 10
includes a gas compressor 12 that is located at the surface of the well for purposes of

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introducing pressurized gas into an annulus 15 of the well. To control the communication of gas between the annulus 15 and a central passageway 17 of the production tubing 14, the system 10 may include several gas lift mandrels 16 (gas lift mandrels 16a, 16b and 16c, depicted as examples). Each one of these gas lift mandrels 16 includes an associated gas lift valve 18 (gas lift valves 18a, 18b and 18c, depicted as examples) that responds to the annulus pressure. More specifically, when the annulus pressure at the gas lift valve 18 exceeds a predefined threshold, the gas lift valve 18 opens to allow communication between the annulus 15 and the central passageway 17. For an annulus pressure below this threshold, the gas lift valve 16 closes and thus, prevents communication between the annulus 15 and the central passageway 17.

It is typically desirable to maximize the number of cycles in which each gas lift valve 18 may be opened and closed, as the cost of the gas lift valves 18 may be a significant component of the overall production costs. The number of times that a gas lift valve may be opened and closed may be a function of the loading that is experienced by the various seals of the gas lift valve 18.

SUMMARY

In an embodiment of the invention, a gas lift valve that is usable with a subterranean well includes a housing, a valve stem and at least one bellows. The housing has a port that is in communication with a first fluid, and the valve stem is responsive to the first fluid to establish a predefined threshold to open the valve. The bellow(s) form a seal between the valve stem and the housing. The bellow(s) are subject to a force that is exerted by the first fluid; and a second fluid contained in the bellow(s) opposes the force that is exerted by the first fluid.

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According to another embodiment of the present invention, there is provided a gas lift valve usable with a subterranean well, comprising: a valve stem movable to control communication between an annulus of the well and a central passageway of a tubing surrounded by the annulus; and a bellows that forms a fluid chamber for receipt of a fluid via a fluid conduit, the bellows being responsive to a pressure inside the central passageway of the tubing to cause fluid in the fluid chamber of the bellows to exert a force on the valve stem to move the stem in a direction associated with the opening of the valve.

Advantages and other features of the invention will become apparent from the following description, drawing and claims.

BRIEF DESCRIPTION OF THE DRAWING

Fig. 1 is a schematic diagram of a gas lift system according to the prior art.

Fig. 2 is a schematic diagram of a portion of a gas lift mandrel according to an embodiment of the invention.

5 Fig. 3 is a schematic diagram of a middle portion of a gas lift valve according to an embodiment of the invention.

Fig. 4 is a schematic diagram of a lower portion of the gas lift valve according to an embodiment of the invention.

10 Figs. 5 and 6 are schematic diagrams of gas lift valves according to other embodiments of the invention.

Fig. 7 is a schematic diagram of a bellows assembly in accordance with another embodiment of the invention.

DETAILED DESCRIPTION

15 Referring to Fig. 2, an embodiment 20 of a gas lift mandrel in accordance with the invention is constructed to be installed in a production tubing (not shown) for purposes of controlling the introduction of gas into a central passageway of the production tubing. As shown, the gas lift mandrel 20 includes two generally cylindrical passageways 22 and 24, each of which has a longitudinal axis that is parallel to the longitudinal axis of the production tubing. More particularly, the passageway 24 is coaxial with the longitudinal axis of the production tubing, as the passageway 24 forms part of the central passageway of the production tubing. The passageway 22 is eccentric to the passageway 24 and houses a gas lift valve 30.

25 The purpose of the gas lift valve 30 is to selectively control fluid communication between an annulus of the well and the central passageway of the production tubing so that gas may be introduced into the production tubing at the location of the gas lift valve 30. The term "annulus" refers to the annular region that surrounds the exterior of the production tubing. For a cased wellbore, the "annulus" may include the annular space, or

region, between the interior surface of the casing string and the exterior surface of the production tubing. The gas lift valve 30 may be part of a gas lift system. In such a

5 system, a gas may be introduced into the well annulus so that one or more of the gas lift valves 30 (that are installed in the production tubing) may be operated for purposes of introducing the gas into the central passageway of the production tubing, as can be appreciated by one skilled in the art.

10 More specifically, the function of the gas lift valve 30 is to control communication between its one or more inlet ports 108 and its one or more output ports 120. The gas lift mandrel 20 includes one or more inlet ports 28 that are in communication with the annulus; and the gas lift valve 30 includes seals (O-rings, MSE seals, or T-seals, for example) 110 that straddle the inlet port(s) 28 and inlet ports 108 for purposes creating a sealed region for the gas lift valve 30 to receive fluid from the annulus. The outlet port(s) 120 are in communication with one or more outlet ports 26 formed in the mandrel 20 between the passageways 22 and 24. Thus, due to this arrangement, when the gas lift valve 30 is open, gas flows from the annulus, through the ports 28, 108, 120 and 26 (in the listed order) and into the passageway 24. When the gas lift valve 30 is closed, the gas lift valve 30 blocks communication between the ports 108 and 120 to isolate the passageway 24 from the annulus.

20 In general, the gas lift valve 30 transitions between its open and closed states in response to annulus or tubing pressure. Typically, if the gas lift valve 30 is an injection pressure operated (IPO) valve it is responsive to annulus pressure. If the gas lift valve 30 is a production pressure operated (PPO) valve, it is typically responsive to tubing pressure. When the annulus or tubing pressure exceeds a predefined threshold, the gas lift valve 30 opens; and otherwise, the gas lift valve 30 closes. In some embodiments of the invention, this predefined threshold may be established by the presence of a gas charge in the gas lift valve 30, as further described below.

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A more specific embodiment of the gas lift valve 30 is illustrated in Figs. 3 and 4. In this manner, Fig. 3 depicts a middle section 30A of the gas lift valve 30, and Fig. 4 depicts a lower section 30B of the gas lift valve.

Referring to Fig. 3, in some embodiments of the invention, the gas lift valve 30 includes a pressure or reservoir 60 that forms part of a gas charge section of the gas lift valve 30, a section that establishes a bias to keep the gas lift valve 30 closed and a predefined annulus threshold that must be overcome to open the valve 30. More specifically, in some embodiments of the invention, the reservoir 60 may be filled with an inert gas, such as Nitrogen, that exists in the reservoir 60 for purposes of exerting a closing force on a gas stem 70 of the gas lift valve 30.

The gas stem 70 and a fluid stem 80 (of the valve 30) collectively form a valve stem for the gas lift valve 30. Assuming the gas lift valve 30 is closed, the valve stem moves in an upward direction to open the gas lift valve 30; and assuming the gas lift valve 30 is open, the valve stem moves in a downward direction to close the gas lift valve 30. More specifically, the gas stem 70 is coaxial with the longitudinal axis 40 of the gas lift valve 30 and is connected at its lower end 70a to the upper end 80b of the fluid stem 80. The fluid stem 80 is also coaxial with the longitudinal axis 40 of the gas lift valve 30. It is noted that the cross-sectional diameters of the gas 70 and fluid 80 stems are different. This relationship permits a lower pressure to be used in the reservoir 60, as further described below.

It is important to note that although the embodiment shown in Fig. 3 shows the gas stem 70 affixed to the fluid stem 80, in alternate embodiments, the gas stem 70 and fluid stem 80 are separated parts that are coupled together by pressure during activation. In further alternate embodiment, the gas stem 70 and the fluid stem 80 are manufactured as a single part. Referring also to Fig. 4, near its lower end 80a, the fluid stem 80 has a ball-type tip 104 that, when the gas lift valve 30 is closed, forms a seal with a valve seat 103 for purposes of closing off communication through a port 102 of the gas lift valve 30.

Because all communication between the inlet 108 and outlet 120 ports occurs through the port 102, the gas lift valve 30 is closed when the tip 104 is seated in the valve seat 103.

This condition occurs when the valve stem is at its point farthest point downward travel. Conversely, the gas lift valve 30 is open when the valve stem is raised and the tip 104 is not seated in the valve seat 103.

Referring to Fig. 3, the gas pressure inside the reservoir 60 acts on a top surface 75 of the gas stem 70 to create a downward force on the valve stem. This downward force, in turn, tends to keep the gas lift valve 30 closed in the absence of a greater opposing force that may be developed by the annulus or tubing pressure on the valve stem (as described below).

The gas reservoir 60 is formed from an upper housing section 79 that contains a chamber 78 (of the gas lift valve 30) for storing the gas in the reservoir 60. The chamber 78 may also house the gas stem 70 and an upper bellows assembly, described below. The upper housing section 79 is connected to a middle housing section 50 of the gas lift valve 30.

The gas lift valve 30 includes an upper bellows assembly that forms a flexible seal between the gas stem 70 and the middle housing section 50 to accommodate movement of the valve stem. In some embodiments of the invention, the upper bellows assembly may include a seal bellows 52 and a compensation bellows 54, both of which are coaxial with and circumscribe the gas stem 70. The seal 52 and compensation 54 bellows are located inside the chamber 78, as depicted in Fig. 3.

As shown, the seal bellows 52 is located closer to the upper end 70b of the gas stem 70 than to the lower end 70a of the gas stem 70; and the seal bellows 52 circumscribes this upper portion of the gas stem 70. The upper end of the seal bellows 52 is connected to the upper end 70b of the gas stem 70, and the lower end of the seal bellows 52 is connected to an annular plate 56.

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The compensation bellows 54 circumscribes the lower part of gas stem 70 and has a larger diameter than the seal bellows 52. The upper end of the compensation bellows 54 is connected to the annular plate 56, as the plate 56 radially extends between the upper end of the compensation bellows 54 and the lower end of the seal bellows 52. The lower
5 end of the compensation bellows 54 is attached to the middle housing section 50.

It should be understood that in alternate embodiments, the relative location of the seal bellows 52 and the compensation bellows 54 along the gas stem 70 can be inverted. For example, the compensation bellows 54 can be located closer to the upper end 70b of the gas stem 70, while the seal bellows circumscribes the lower part of the gas stem 70.

10 In the embodiment shown, when the gas stem 70 (and thus, the valve stem) moves in a downward direction, the compensation bellows 54 longitudinally expands and the seal bellows 52 longitudinally compresses. Conversely, when the gas stem 70 moves in an upward direction, the compensation bellows 54 longitudinally compresses and the seal bellows 52 longitudinally expands.

15 The pressure that is exerted on the bellows 52 and 54 by the gas inside the reservoir 60 may cause a significant pressure differential across the walls of the seal bellows 52 and across the walls of the compensation bellows 54, if not for the pressure balancing features of the gas lift valve 30. In some embodiments of the invention, the pressure balancing features include an incompressible fluid that is contained inside the
20 bellows 52 and 54.

More specifically, in some embodiments of the invention, the incompressible fluid is contained within annular spaces 62 and 63. The walls of the seal bellows 52 define the annular region 62, a region that is located between the interior surface of the seal bellows 52 and the adjacent exterior surface of the gas stem 70. The walls of the compensation
25 bellows 54 define the annular region 63, a region that is located between the interior surface of the seal bellows 54 and the adjacent exterior surface of the gas stem 70. The two regions 62 and 63 are isolated by the bellows 52 and 54 from the gas in the reservoir

60 and are in communication so that the incompressible fluid may move between the regions 62 and 63 when the bellows 52 and 54 are compressed/decompressed.

5 The incompressible fluid serves to remove any pressure differential that otherwise exists across the walls of the bellows 52 and 54 due to the pressure that is exerted by the gas in the reservoir 60. More specifically, the incompressible fluid is a non-compressible fluid that exerts forces (on the interior surface of the walls of the bellows 52 and 54) that are equal and opposed to the forces on the outer surfaces of the walls of the bellows 52 and 54 (exerted by the gas in the reservoir 60).

10 In operation, when the gas stem 70 moves in a downward direction, the compensation bellows 54 expands and the seal bellows 52 compresses. Therefore, some of the incompressible fluid contained within the seal bellows 52 is displaced into the compensation bellows 54, as the volume of incompressible fluid remains constant. When the gas stem 70 moves in an upward direction, the compensation bellows 54 compresses and the seals bellows 52 expands. Some of the incompressible fluid contained within the compensation bellows 54 is displaced into the seal bellows 52, as the volume of the incompressible fluid remains constant. Thus, regardless of the positions of the bellows 52 and 54, the incompressible fluid remains inside the bellows 52 and 54 to compensate forces that are exerted by the gas inside the reservoir 60.

20 To summarize, the bellows 52 and 54 and the incompressible fluid establish a pressure compensation system to equalize the pressure difference across the walls of the bellows 52 and 54. The result is that the bellows 52 and 54 transfer a more uniform load to the incompressible fluid, and consequently to the seal 76.

25 Among the other features of the gas charge section of the gas lift valve 30, the gas lift valve 30 may include, in some embodiments of the invention, a fluid fill port 74 for purposes of introducing the incompressible fluid into the annular regions 62 and 63. The fill port 74 may be located, for example, in the top surface of the gas stem 70 and may be in communication with the annular regions 62 and 63 via one or more passageways 77

that are formed in the gas stem 70. The gas lift valve 30 also includes an annular seal 76 that closely circumscribes the exterior surface of the gas stem 70 to form a seal between the annular regions 62 and 63 and the middle housing section 50 for purposes of sealing the incompressible fluid inside the bellows 52 and 54. The gas lift valve 30 also includes
5 another annular seal 82 for purposes of forming a seal between the exterior surface of the fluid stem 80 and the incompressible fluid used for purposes of equalizing, or balancing, pressures that are exerted on bellows on the well fluid section part of the gas lift valve, described below.

Turning to the well fluid section of the gas lift valve 30, in some embodiments of
10 the invention, this section includes a lower bellows assembly. This lower bellows assembly includes an upper seal bellows 84 and a lower compensation bellows 86, both of which are coaxial with the longitudinal axis 40 of the gas lift valve 30. The seal bellows 84 has a top end 84a that is connected to the fluid stem 80. A radially extending annular plate 88 connects the lower end 84b of the seal bellows 84 to the upper end 86a
15 of the compensation bellows 86. The lower end 86b of the compensation bellows 86, in turn, is connected to the middle housing section 50. As discussed above with regard to the upper bellows assembly, in alternate embodiments, the orientation of the upper seal bellows 84 and the lower compensation bellows 86 can be reversed.

As depicted in Fig. 3, the seal bellows 84 circumscribes part of the fluid stem 80
20 and has a smaller diameter than the diameter of the compensation bellows 86. The compensation bellows 86 circumscribes a lower portion of the fluid stem 80.

Fluid from the well annulus is in communication with an annular region 90 that exists between the exterior surface of the fluid stem 80 and the interior wall surfaces of the bellows 84 and 86. This annular region 90 is in communication with a fluid chamber
25 83 formed in a lower housing section 81 of the gas lift valve 30. The lower housing section 81 is connected to the middle housing section 50, and in addition to establishing

the fluid chamber 83, the lower housing section 81 contains the lower bellows assembly and fluid stem 80.

5 An annular region 92 exists between the outer surface of the wall of the seal bellows 84 and the inner surface of the middle housing 50; and an annular region 91 exists between the outer surface of the wall of the compensation bellows 86 and the inner surface of the middle housing 50. Both regions 91 and 92 contain the incompressible fluid for purposes of equalizing the pressure across the walls of the bellows 84 and 86, in a similar arrangement to that described for the bellows 52 and 54 with the exception that here, the incompressible fluid is located outside of the bellows walls and the fluid that
10 exerts the forces on the bellows walls is located inside of the bellows walls.

In operation, when the fluid stem 80 moves in a downward direction, the bellows 84 compresses, thereby evacuating the incompressible fluid from the annular region 91 into the annular region 92. During the compression of the bellows 84, the bellows 86 expands to compensate the incompressible fluid that is displaced from the compressed
15 annular region 91. Conversely, when the fluid stem 80 moves in an upward direction, the bellows 86 compresses, and fluid that is displaced from the region 92 enters the region 91 as the bellows 84 expands. By maintaining a constant volume of the incompressible fluid, the differential pressure across the walls of the bellows 84 and 86 is eliminated.

20 As described above, the pressure of the gas in the reservoir 60 tends to force the valve stem (i.e., the gas 70 and fluid 80 stems) in a downward direction. However, the pressure that is exerted by fluid in the annulus of the well exerts an upward force on the gas 70 and fluid 80 stems, tending to push the stems 70 and 80 in an upward direction.

Therefore, the pressure inside the reservoir 60 establishes a predefined threshold that must be overcome for the gas stem 70 and the fluid stem 80 to move in an upward
25 direction to open the gas lift valve 30.

In some embodiments of the invention, the diameter of the seal 76 of the gas stem 70 is larger than the diameter of the seal 82 of the fluid stem 80. This means that for a

given pressure level for the reservoir 60, more downward force is developed on the valve stem than the upward force that is developed on the valve stem for the same pressure

level for the annulus fluid. Thus, the above-described relationship of seal diameters between the gas 70 and fluid 80 stems intensifies the pressure that is exerted by the gas in the reservoir 60 with respect to the pressure that is exerted by the annulus or tubing fluid. Such intensifier relationship enables the use of lower charge pressure based on a given annulus or tubing pressure.

Referring to Fig. 4, among its other features, in some embodiments of the invention, the gas lift valve 30 includes the radial ports 108 (see also Fig. 2) that are formed in the lower housing section 81 for purposes of establishing fluid communication between the annulus and the fluid chamber 83. The bottom end of the valve stem, i.e., the tip 104, controls communication of the annulus fluid through the port 102, a port that establishes communication between the fluid chamber 83 and an intermediate chamber 103. Thus, when the gas 70 and fluid 80 stems are retracted in an upward direction, the pin 104 is moved off of the valve seat 103 to permit fluid communication between the chambers 83 and 103.

A one-way communication path exists between the intermediate chamber 103 and an exit chamber 105, a chamber 105 in which the outlet ports 120 (see also Fig. 2) are formed. In this manner the one-way communication path is effectively established by a check valve, a valve that ensures that annulus fluid flows from the chamber 103 into the production tubing and does not flow from the production tubing into the annulus.

The check valve opens in response to annulus pressure so that fluid flows from the annulus through a port 119 that exists between the chambers 103 and 105. In some embodiments of the invention, the check valve may include a valve stem 118 that has a tip 121 that seats in a valve seat 123 for purposes of preventing fluid from flowing in the reverse direction through the port 119. Thus, a differential force that would cause fluid to flow from the production tubing into the annulus forces the tip 121 into the valve seat 123

to block communication through the port 119. Conversely, a differential force that would cause fluid to flow from the annulus into the production tubing removes the tip 121 from the valve seat 123 to permit communication through the port 119

Referring to Fig. 5, in some embodiments of the invention, the gas lift valve 30 may be replaced by a gas lift valve 200. Components (of the gas lift valve 200) that are similar to components of the gas lift valve 30 are denoted by similar reference numerals.

Unlike the gas lift valve 30, the gas lift valve 200 includes a tubing pressure assist mechanism for purposes of using pressure in the central passageway of the production tubing to assist in opening the gas lift valve 200. Such a system may be beneficial when a relatively lower pressure is used in the annulus for purposes of opening the gas lift valve.

More specifically, in some embodiments of the invention, the gas lift valve 200 includes a tubing assist bellows 202 that is in communication with the central passageway of the production tubing so that the tubing pressure compresses the bellows 202. The exterior of the bellows 202 is in communication with a port 201 that, in turn, communicates with the tubing fluid.

The bellows 202 contains a fluid (an incompressible fluid, for example) that is in communication (via a communication line 209) to an interior space of another bellows 210. The bellows 210, in turn, is connected to a valve stem 212 so that when the bellows 202 compresses (due to the force exerted due to the tubing pressure), the fluid enters the bellows 210 to expand the bellows 210. This expansion, in turn, lifts the stem 212 to open the gas lift valve 200 to allow communication between the well annulus and the production tubing.

The tendency of the bellows 210 to expand and open the gas lift valve 30 in response to the tubing pressure is countered by a charge pressure that exists inside an internal charge reservoir 206 of the valve 200. In this manner, the bellows 210 is contained inside the reservoir 206 so that the gas inside the reservoir 206 exerts a force on the exterior surface of the bellows 210. Thus, the predefined threshold established by the

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charge 206 must be overcome to allow the bellows 210 to expand by a sufficient amount to limit the stem 212 to lift the stem 212 to open the gas lift valve 200.

In some embodiments of the invention, the charge reservoir 206 is in communication (via a pressure line 215) to a space inside another bellows 220. In this manner, gas from the reservoir 206 may work to expand the bellows 220. When expanded, the bellows 220 tends to move the stem 212 in a downward direction to close the gas lift valve 200. However, the tendency of the bellows 220 to expand is countered by pressure in the well annulus. In this regard, the exterior of the bellows 220 is in communication with the well annulus via radial inlet ports 108.

In some embodiments of the invention, the gas lift valves 30 and 200 may be replaced by a gas lift valve 300 that is depicted in Fig. 6. Components (of the gas lift valve 300) that are similar to components of the gas lift valves 30 and 200 are denoted by similar reference numerals.

Unlike the gas lift valves described above, the gas lift valve 300 includes a venturi orifice 326 between the ports 102 and 119 for purposes of minimizing the pressure drop and the turbulence in the flow of gas from the well annulus to the central passageway of the production tubing.

Other embodiments are within the scope of the following claims. For example, in the embodiments described above, for each set of seal and compensation bellows, a seal (seals 76 and 82, for example) was located in the body, or housing, of the gas lift valve assembly to form a seal between a rod, or stem (stems 70 and 80, for example) and the housing. This arrangement kept the volume of incompressible fluid contained within the bellows constant. However, in other embodiments of the invention, the seal may be located in, or secured to, the rod so that the seal moves with the rod.

As a more specific example, Fig. 7 depicts an exemplary bellows assembly 350 according to another embodiment of the invention. The assembly 350 includes a seal bellows 354, a compensation bellows 356 and a stem, or rod 352, to expand and

compress the bellows 354 and 356, as described above in the other embodiments described herein. However, unlike these other embodiments, a seal 360 (an O-ring seal, MSE seal, or T-seal, for example) is attached to, or located in, the rod 352 so that the seal 360 moves with the rod 352.

5 More particularly, the seal 360 is located inside an annular groove 362 of the rod 352 and forms a seal between the exterior surface of the rod and an interior surface of a housing 370. This interior surface of the housing 370 defines a passageway 364 through which the rod 352 slides. The seal 360 maintains an incompressible fluid 380 within the interior regions defined by the seal 354 and compensation 356 bellows.

10 Unlike the embodiments in which the seal is located in the housing, the seal 360 in the assembly 350 moves with the rod 352. This arrangement affects the movement of the bellows 354 and 356, since the movement of the seal 360 with the rod 352 forces the volume of fluid 380 into the interior regions that are defined by the bellows 354 and 356. In response to the rod 352 moving in an upward direction, the seal 354 and
15 compensation 356 bellows move in upward directions. The rates at which the seal 354 and compensation 356 bellows move is different.

 Thus, by placing the seal 360 on the rod 352, the movement of the bellows 352 and 354 follows the movement of the rod 352. The internal regions that are defined by the seal 354 and compensation 356 bellows is still filled with the incompressible fluid
20 380 that transfers the pressure loads to the seal 360, allowing the bellows to see no differential loading.

 In the preceding description, directional terms, such as "upper," "lower," "vertical," "horizontal," etc. may have been used for reasons of convenience to describe the gas lift valve and its associated components. However, such orientations are not
25 needed to practice the invention, and thus, other orientations are possible in other embodiments of the invention. For example, the gas lift valve and its associated

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components, in some embodiments of the invention, may be tilted by approximately 90° to the orientations depicted in the figures.

While the present invention has been described with respect to a limited number of embodiments, those skilled in the art, having the benefit of this disclosure, will appreciate numerous modifications and variations therefrom. It is intended that the appended claims cover all such modifications and variations as fall within the scope of this present invention.

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CLAIMS:

1. A gas lift valve usable with a subterranean well, comprising:
a valve stem movable to control communication between an annulus of the well and a central passageway of a tubing surrounded by the annulus; and
5 a bellows that forms a fluid chamber for receipt of a fluid via a fluid conduit, the bellows being responsive to a pressure inside the central passageway of the tubing to cause fluid in the fluid chamber of the bellows to exert a force on the valve stem to move the stem in a direction associated with the opening of the valve.
2. The valve of claim 1, further comprising:
10 another bellows; and
the fluid being a fluid communicated between the first bellow and said another bellows,
wherein the second bellow exerts said force on the stem in response to the communication of the fluid between the first bellows and said another bellows.
- 15 3. The valve of claim 1, further comprising a fluid seal element located between the valve stem and the housing.
4. The valve of claim 3, wherein the fluid seal element comprises an O-ring.
5. The valve of claim 3, wherein the fluid seal element is adapted to move with the valve stem.
- 20 6. The valve of claim 3, wherein the fluid seal element is secured to the housing.

FIG. 1
(Prior Art)

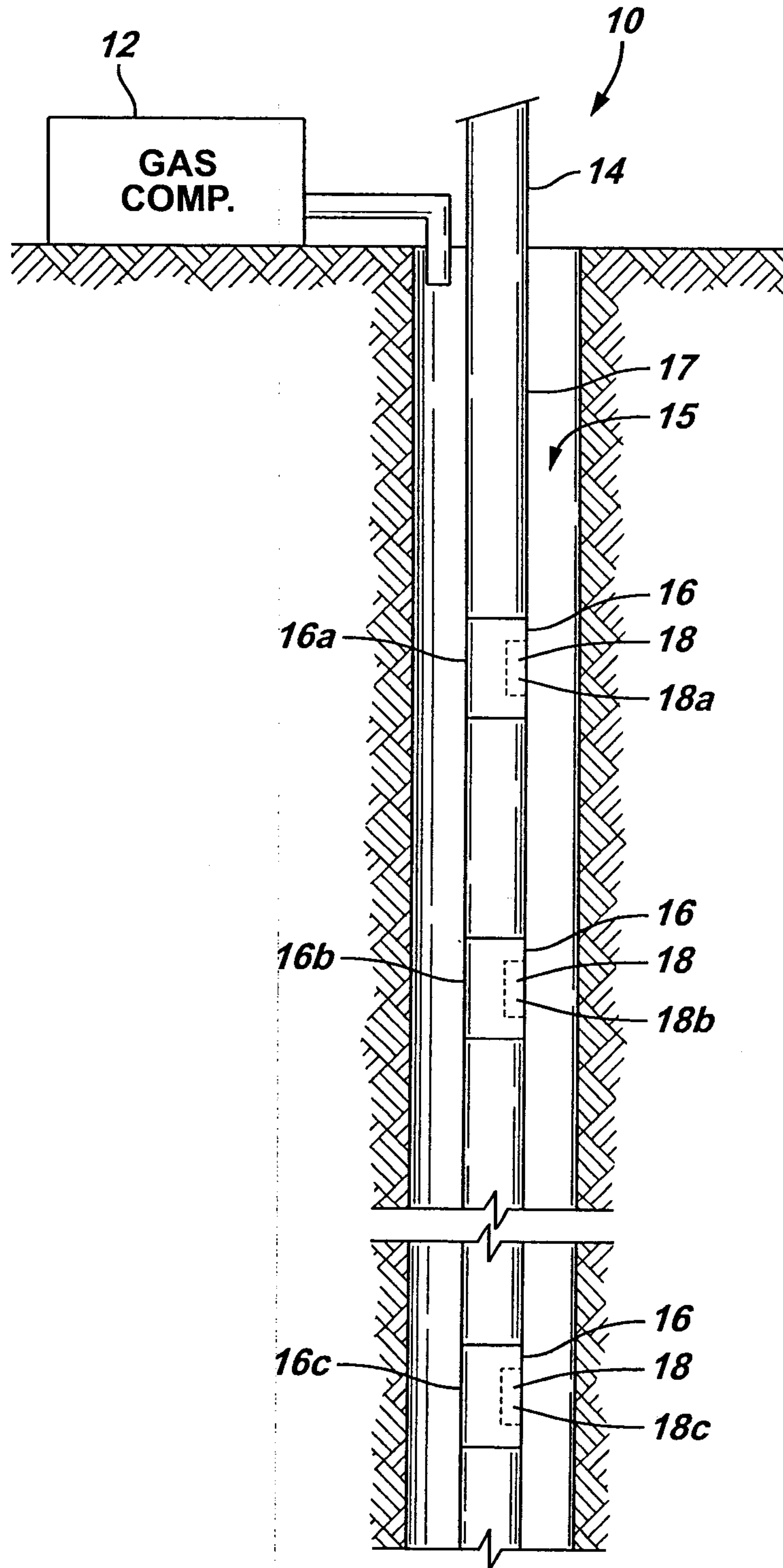
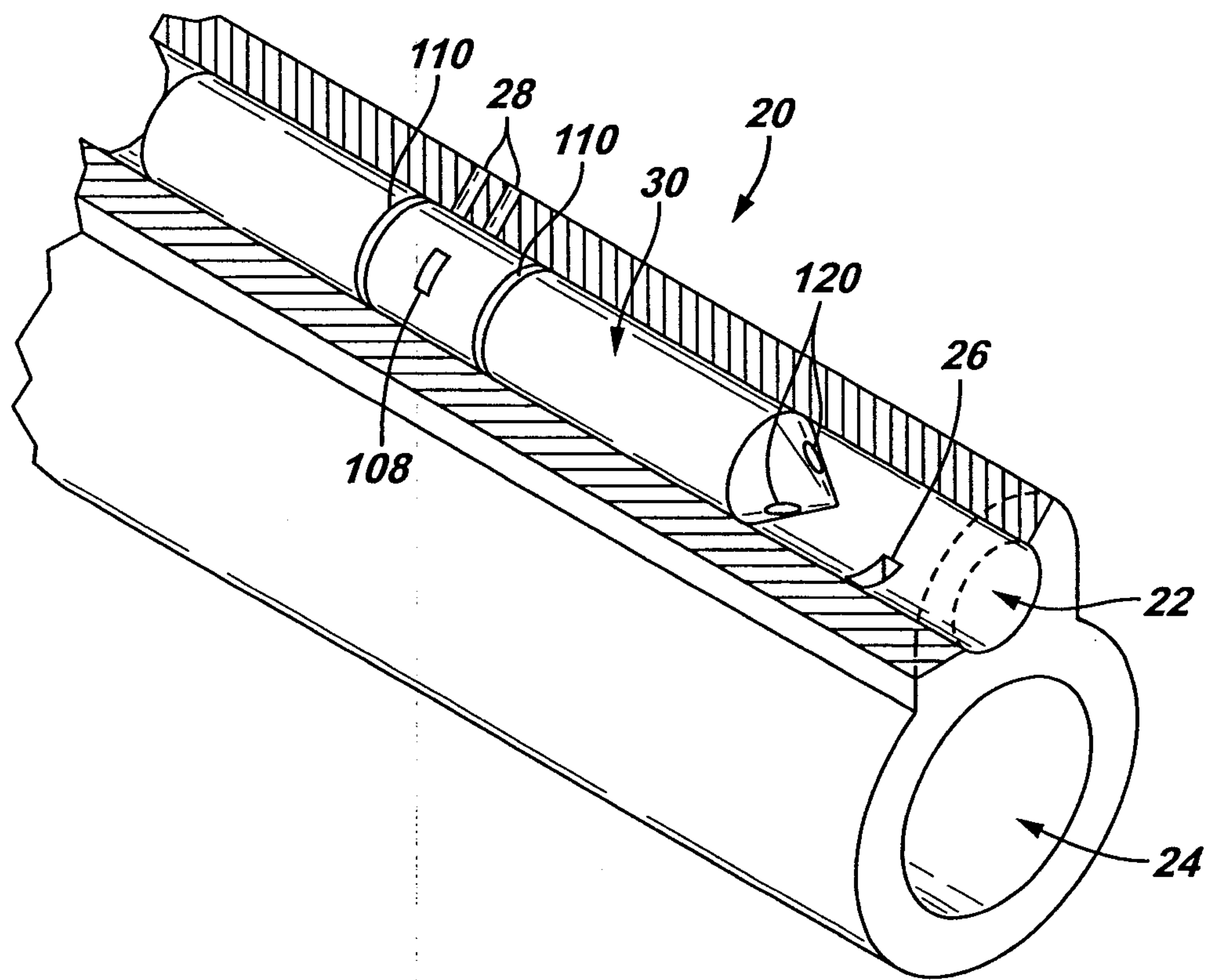


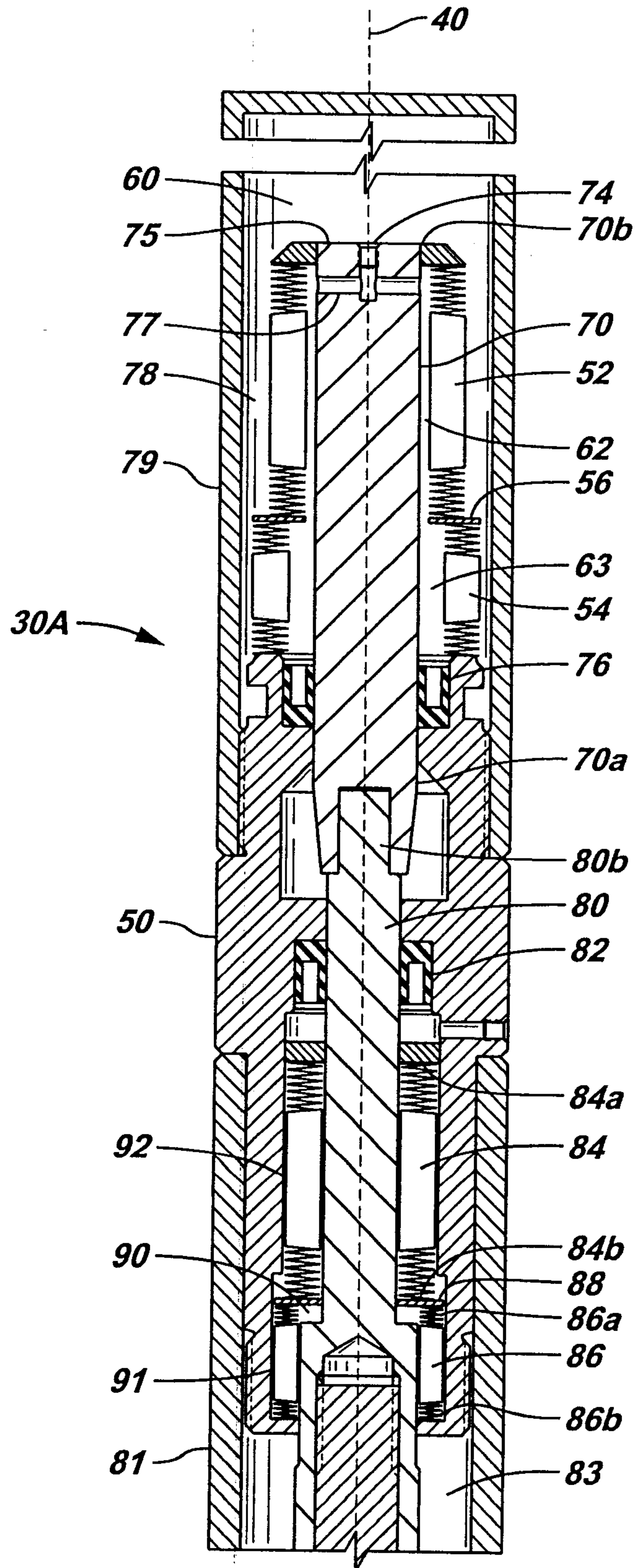
FIG. 2



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FIG. 3



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FIG. 4

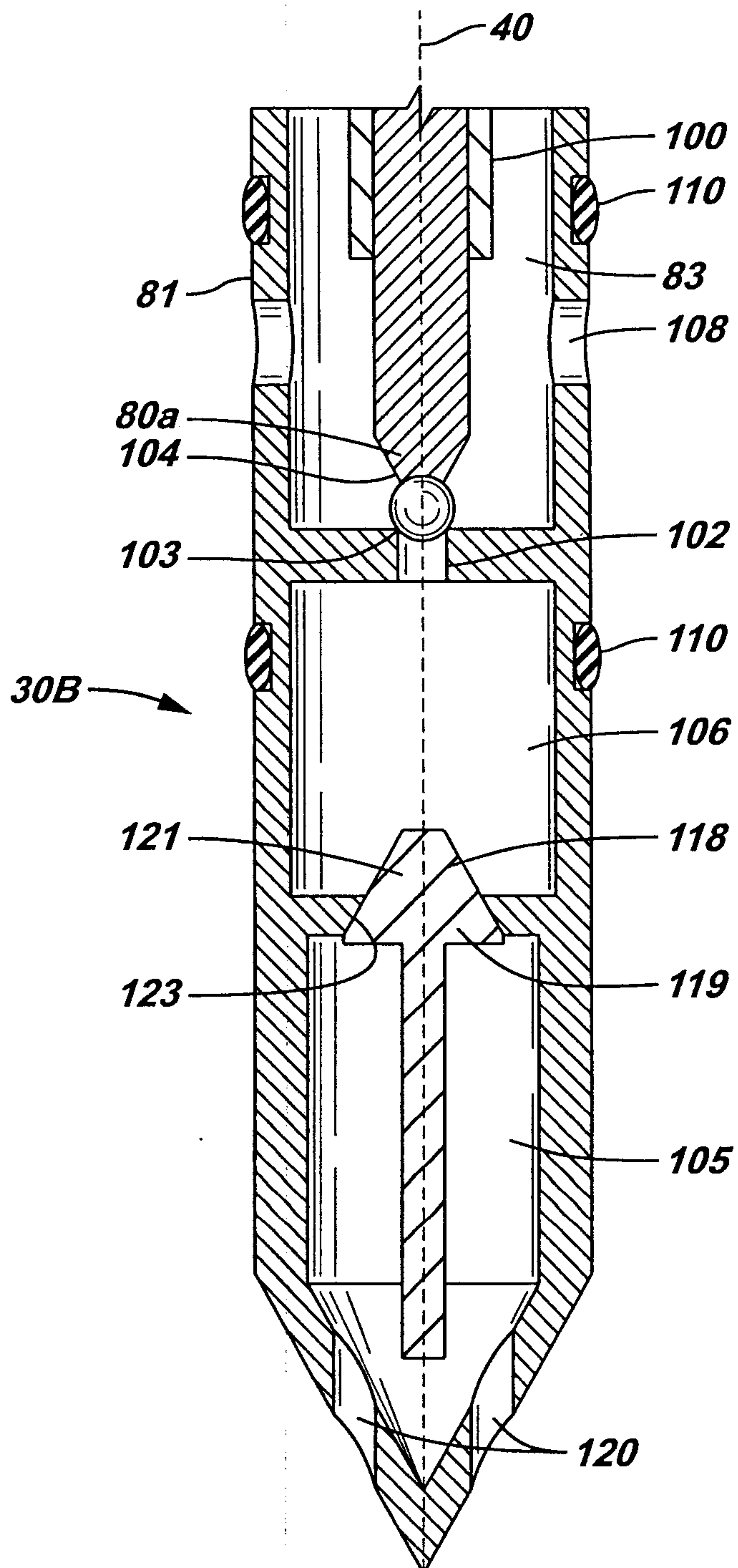


FIG. 5

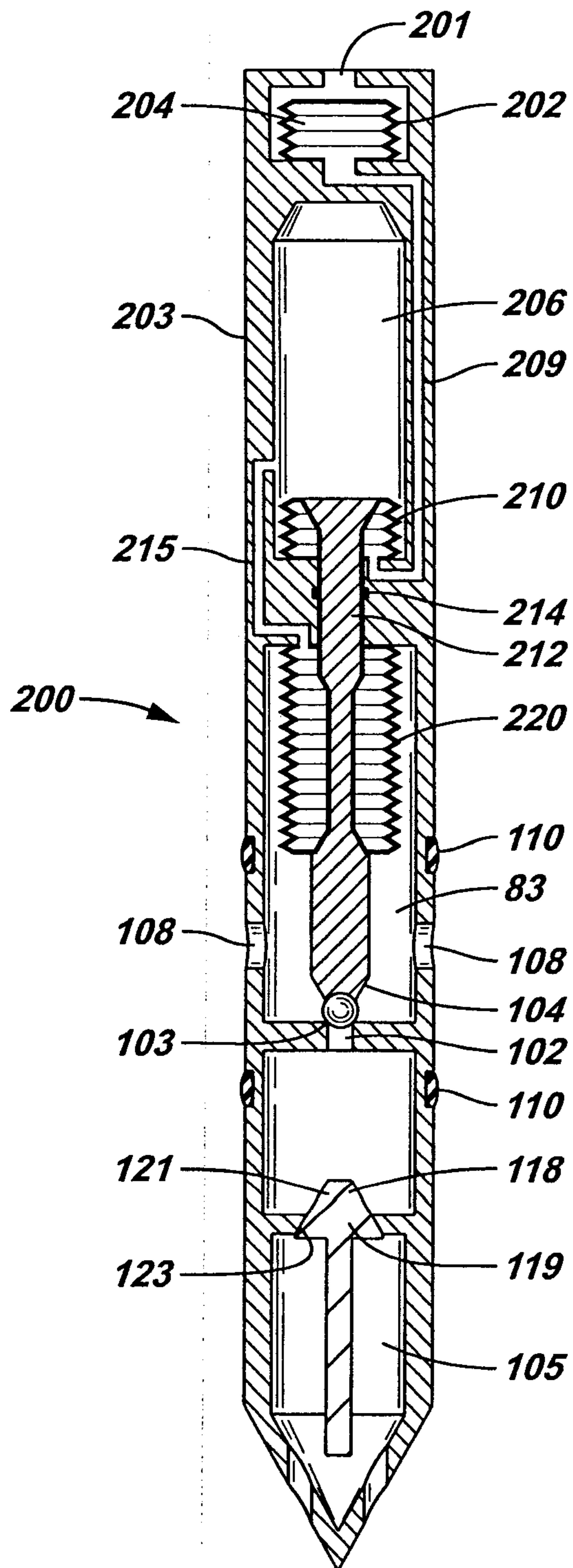


FIG. 6

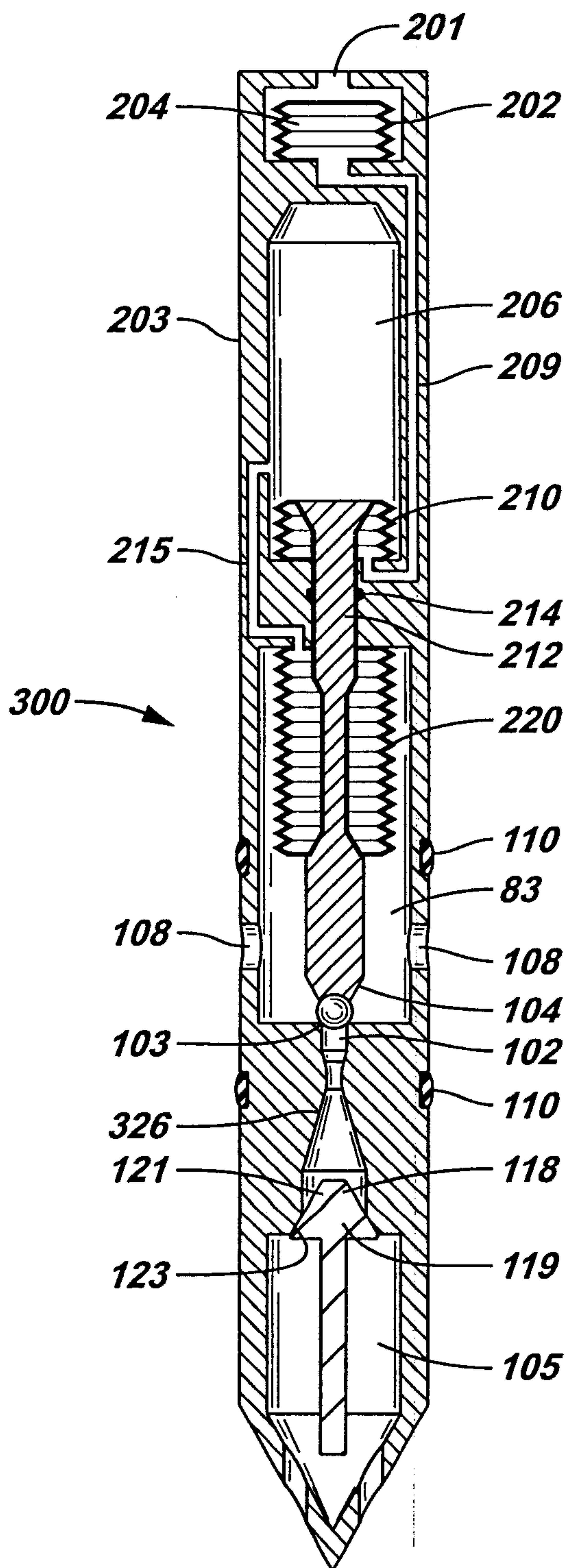


FIG. 7

