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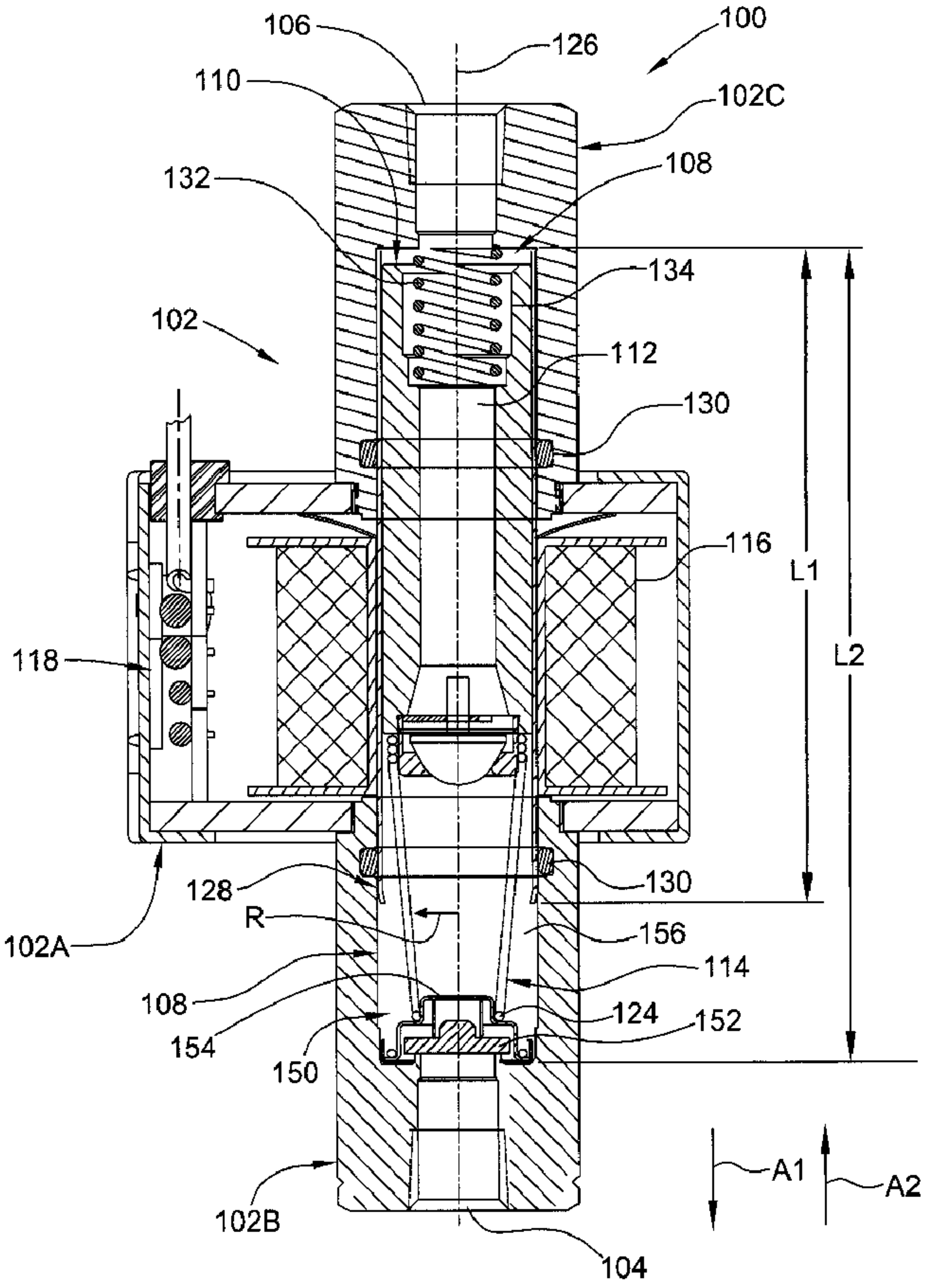
(54) **Titre : POMPE A SOLENOIDE HAUTE PRESSION**
(54) **Title: HIGH PRESSURE SOLENOID PUMP**



(57) **Abrégé/Abstract:**

A solenoid pump, including: an inlet port, an outlet port, and a first through-bore connecting the inlet and outlet ports; a plunger disposed within the first through-bore and including a second through-bore; a spring arranged to urge the plunger toward the outlet

(57) **Abrégé(suite)/Abstract(continued):**
port; a solenoid coil disposed about a portion of the plunger and arranged to displace the plunger toward the inlet port in response to coil power applied to the solenoid coil; and a control unit for: accepting an input voltage; generating the coil power during an interval equal to a first time period; supplying the coil power to the solenoid coil; and selecting a duration of the first time period such that the duration of the first time period varies according to the input voltage.

ABSTRACT

A solenoid pump, including: an inlet port, an outlet port, and a first through-bore connecting the inlet and outlet ports; a plunger disposed within the first through-bore and including a second through-bore; a spring arranged to urge the plunger toward the outlet port; a solenoid coil disposed about a portion of the plunger and arranged to displace the plunger toward the inlet port in response to coil power applied to the solenoid coil; and a control unit for: accepting an input voltage; generating the coil power during an interval equal to a first time period; supplying the coil power to the solenoid coil; and selecting a duration of the first time period such that the duration of the first time period varies according to the input voltage.

HIGH PRESSURE SOLENOID PUMP

FIELD OF THE INVENTION

[0001] The invention relates generally to a solenoid pump with a conical, variable rate spring to enable maximum displacement of a plunger in the pump and to increase back pressure values under which the pump can operate. The invention also generally relates to a control scheme for a solenoid pump that varies a duty cycle according an input voltage used to power the pump.

BACKGROUND OF THE INVENTION

[0002] Known solenoid pumps use linear springs to bias a plunger against displacement by a solenoid coil in a pumping cycle. When the springs are fully compressed, the springs occupy an undesirably large space since the coils for the springs stack upon each other. Known control schemes for solenoid pumps use a fixed duty cycle, typically 50, regardless of the magnitude of the input voltage to be used to energize the solenoid coils for the pumps. As a result, too little power is delivered to the coils for low values of the input voltage and the coils remain energized even after plungers for the pumps have fully displaced to fully compress the springs for the pumps. As a result, the pumps consume unnecessarily high amounts of energy and undesirable amounts of heat are generated, which degrades operation of the pumps.

[0003] Typically, back pressure is present at the outlet port of a solenoid pump and limits operation of the pump, that is, the pump can operate only up to a certain back pressure level. In general, the back pressure works against the spring used to bias the plunger. For example, when the back pressure is greater than the biasing force of the spring, the pumping cycle is terminated (the plunger cannot return to a “rest” position when the coil is de-energized). The known use of linear springs limits the back pressure under which known solenoid pumps can operate. The spring biasing force must be relatively lower to enable the initiation of the plunger displacement when the coil is energized. Since the spring is linear, only the same relatively lower biasing force is available to counteract the back pressure. Known solenoid pumps cannot operate with a backpressure over about 10 psi.

[0004] Common rail systems use a relatively low pressure pump to pump fuel from a fuel source to a high pressure pump. The high pressure pump supplies fuel from the low pressure pump to a distribution line, for example, a distribution pipe feeding fuel injectors for an engine. The high pressure pump in a common rail system can operate at pressures of over 29,000 psi. A pressure regulating valve placed between the low and high pressure pumps typically creates a back pressure on the outlet port of the low pressure pump greater than the 10 psi maximum backpressure under which known solenoid pumps can operate. Thus, known common rail systems teach the use of pumps other than solenoid pumps.

SUMMARY OF THE INVENTION

[0005] According to aspects illustrated herein, there is provided a control unit for a solenoid pump including: an inlet port, an outlet port, and a first through-bore connecting the inlet and outlet ports; a plunger disposed within the first through-bore and including a second through-bore; a spring arranged to urge the plunger toward the outlet port; a solenoid coil disposed about a portion of the plunger and arranged to displace the plunger toward the inlet port in response to coil power applied to the solenoid coil, the control unit including: an input for accepting an input voltage; and a power circuit for: generating the coil power during an interval equal to a time period; supplying the coil power to the solenoid coil; and selecting a duration of the time period such that the duration of the time period varies according to the input voltage.

[0006] According to aspects illustrated herein, there is provided a solenoid pump, including: an inlet port, an outlet port, and a first through-bore connecting the inlet and outlet ports; a plunger disposed within the first through-bore and including a second through-bore; a spring arranged to urge the plunger toward the outlet port; a solenoid coil disposed about a portion of the plunger and arranged to displace the plunger toward the inlet port in response to coil power applied to the solenoid coil; and a control unit for: accepting an input voltage; generating the coil power during an interval equal to a first time period; supplying the coil power to the solenoid coil; and selecting a duration of the first time period such that the duration of the first time period varies according to the input voltage.

[0007] According to aspects illustrated herein, there is provided a solenoid pump, including: a housing with an inlet port and an outlet port; a first through-bore connecting the inlet and outlet ports; a plunger disposed within the first through-bore and including a second through-bore; a spring arranged to urge the plunger toward the outlet port; a solenoid coil arranged to displace the plunger toward the inlet port in response to a coil power applied to the solenoid coil; and a control unit for controlling operation of the solenoid coil such that when the solenoid coil is energized by the coil power to displace the plunger and the spring is fully compressed by the plunger, coils forming the spring are aligned in a direction orthogonal to a longitudinal axis passing through the inlet and outlet ports.

[0008] According to aspects illustrated herein, there is provided a solenoid pump, including: a housing with an inlet port and an outlet port; a first through-bore connecting the inlet and outlet ports; a sleeve disposed within the first through-bore and displaceable parallel to a longitudinal axis passing through the inlet and outlet ports; a plunger disposed within the first through-bore, displaceable parallel to the longitudinal axis, and including a second through-bore; a spring arranged to urge the plunger toward the outlet port; a solenoid coil arranged to displace the plunger toward the inlet port in response to a coil power applied to the solenoid coil; and a control unit for controlling operation of the solenoid coil such that fluid is transferred from the inlet port to the outlet port through the second through bore.

[0009] According to aspects illustrated herein, there is provided a method of operating a control unit for a solenoid pump including: an inlet port, an outlet port, and a first through-bore connecting the inlet and outlet ports; a plunger disposed within the first through-bore and including a second through-bore; a spring arranged to urge the plunger toward the outlet port; a solenoid coil disposed about a portion of the plunger and arranged to displace the plunger toward the inlet port in response to coil power applied to the solenoid coil, the method including: using an input to accept an input voltage; and using a power circuit to: generate the coil power during an interval equal to a time period; supply the coil power to the solenoid coil; and select a duration of the time period such that the duration of the time period varies according to the input voltage.

[0010] According to aspects illustrated herein, there is provided a method of pumping fluid using a solenoid pump including: an inlet port, an outlet port, and a first through-bore connecting the inlet and outlet ports; a plunger disposed within the first through-bore and including a second through-bore; a spring; a solenoid coil disposed about a portion of the valve assembly; and a control unit. The method includes: urging, using the spring, the plunger toward the outlet port; and using the control unit to: accept an input voltage; determine a magnitude of the input voltage; select a duration of a first time period such that the duration of the first time period varies according to the input voltage; generating, using the input voltage, a coil power during an interval equal to the first time period; supplying the coil power to the solenoid coil such that the plunger displaces toward the inlet port; remove the coil power such that the spring displaces the plunger toward the outlet port.

[0011] According to aspects illustrated herein, there is provided a method of pumping fluid using a solenoid pump including: a housing with an inlet port and an outlet port; a first through-bore connecting the inlet and outlet ports; a plunger disposed within the first through-bore and including a second through-bore; a spring; a solenoid coil; and a control unit. The method including: urging the plunger toward the outlet port with the spring; and using the control unit to apply a coil power to the solenoid coil to displace the plunger toward the inlet port such that the spring is fully compressed by the plunger, and coils forming the spring are aligned in a direction orthogonal to a longitudinal axis passing through the inlet and outlet ports.

BRIEF DESCRIPTION OF THE DRAWINGS

[0012] The nature and mode of operation of the present invention will now be more fully described in the following detailed description of the invention taken with the accompanying drawing figures, in which:

Figure 1 is a plan view of a high pressure solenoid pump;

Figure 2 is a side view of the pump shown in Figure 1;

Figure 3 is an exploded view of the high pressure solenoid pump shown in Figure 1;

Figures 4A – 4C are respective cross-sectional views of the high pressure solenoid pump shown in Figure 1 generally along line 4-4 in Figure 1, depicting various stages of a pumping cycle;

Figure 5A is a table showing duty cycle data for a solenoid pump using a control scheme varying a time for generating coil power;

Figure 5B is a table for a prior art control scheme with a fixed duty cycle;

Figure 6 depicts an exemplary power circuit for a control scheme varying a time for generating coil power according to input voltage.

DETAILED DESCRIPTION OF THE INVENTION

[0013] At the outset, it should be appreciated that like drawing numbers on different drawing views identify identical, or functionally similar, structural elements of the invention. It is to be understood that the invention as claimed is not limited to the disclosed aspects.

[0014] Furthermore, it is understood that this invention is not limited to the particular methodology, materials and modifications described and as such may, of course, vary. It is also understood that the terminology used herein is for the purpose of describing particular aspects only, and is not intended to limit the scope of the present invention, which is limited only by the appended claims.

[0015] Unless defined otherwise, all technical and scientific terms used herein have the same meaning as commonly understood to one of ordinary skill in the art to which this invention belongs. Although any methods, devices or materials similar or equivalent to those described herein can be used in the practice or testing of the invention, exemplary methods, devices, and materials are now described.

[0016] Figure 1 is a plan view of high pressure solenoid pump **100**.

[0017] Figure 2 is a side view of pump **100** shown in Figure 1.

[0018] Figure 3 is an exploded view of high pressure solenoid pump **100** shown in Figure 1.

[0019] Figures 4A – 4C are respective cross-sectional views of high pressure solenoid pump **100** shown in Figure 1 generally along line 4-4 in Figure 1, depicting various stages of a pumping cycle. The following should be viewed in light of Figures 1 through 4C. Pump **100** includes housing **102** with inlet port **104** and outlet port **106**. In an example embodiment, housing **102** is formed by main housing **102A**, inlet housing **102B**, and outlet housing **102C**. Housings **102B** and **102C** are connected to the main housing by any means known in the art, for example, threads. Pump **100** includes through-bore **108** connecting the inlet and outlet ports, and plunger **110** disposed within through-bore **108** and including through-bore **112**. Pump **100** includes spring **114** arranged to urge the plunger toward the outlet port, solenoid coil **116** arranged to displace the plunger toward the inlet port in response to a coil power applied to the solenoid coil, and control unit **118** for controlling operation of the solenoid coil.

[0020] Spring **114** is a variable rate spring. By "variable rate spring" we mean that resistance of the spring to compression of the spring in direction **A1** toward the inlet port increases as the spring is compressed in direction **A1**, for example, by the plunger. Stated otherwise, referring to Hooke's Law: $F = -kx$, the constant k for the spring increases as the spring is compressed. Thus, the further the spring is compressed, the more force is needed to continuing compressing the spring. For example, when the plunger begins displacing in direction **A1** from the position shown in Figure 4A, a certain amount of force is required to compress the spring. As the plunger continues to displace to the position shown in Figure 4B, an increasingly greater amount of force is required to continue compressing the spring. The rate for spring **114** may vary according to pump type and the pressure output of the pump, for example, k for the spring can be varied.

[0021] Spring **114** has a conical shape, for example, diameter **D1** at end **120** of the spring closest to the inlet port in Figure 4A is less than diameter **D2** at end **122** of the spring, opposite end **120**. Thus, when the spring is compressed as shown in Figure 4B, compressed coils **124** forming the spring are aligned in direction **R** orthogonal to longitudinal axis **126** passing through the inlet and outlet ports.

[0022] In an example embodiment, the pump includes sleeve **128** disposed within through-bore **108** and displaceable parallel to axis **126**. The plunger is disposed within the sleeve and in an example embodiment is displaceable within the sleeve parallel to the longitudinal axis. Seals **130**, for example, O-rings, provide a seal between housing **102** and the sleeve, while enabling movement of the sleeve within bore **108**. Length **L1** of the sleeve is less than length **L2** of through bore **108**, thus, the sleeve “floats” within bore **108**. Advantageously, having sleeve **128** “float” within bore **108** increases the ease of fabrication of pump **100**, since fabrication steps that would be needed to fix the sleeve within the pump are eliminated. Further, having the sleeve float enables greater flexibility since sleeves with different lengths **L1** can be easily installed. Also, since **L1** is less than **L2**, tolerances for **L1** can be relaxed, reducing manufacturing cost and complexity. In an example embodiment, sleeve **128** is made from a non-magnetic material.

[0023] The following provides further example detail regarding pump **100** and an example operation of pump **100**. The plunger is arranged to pass fluid through through-bore **112** and longitudinally traverses the pump between the inlet and outlet ports. In an example embodiment, bumper spring **132** is disposed in end **134** of the plunger. The bumper spring contacts shoulder **136** in the housing to cushion the impact of the plunger as the plunger moves from the position of Figure 4B to the fully retracted position of Figure 4A. Sleeve **128** serves as the primary location wherein mechanical pumping operations are performed. Suction valve assembly **138** is disposed at end **140** of the plunger. In an example embodiment, the suction valve assembly includes cap **142**, seat **144**, and stem **146** passing through retainer element **148**. The operation of the suction valve assembly is further described below.

[0024] Pump **100** includes one-way check valve **150**. The check valve enables fluid flow through the inlet port toward the outlet port in direction **A2** and blocks fluid flow in the opposite direction, **A1**. In an example embodiment, the check valve includes sealing element **152** within valve housing **154**. The sealing element seals against the housing, for example, inlet housing **102B** to block flow out of the pump through the inlet port. For example, the one-way check valve is used as part of drawing fuel from a fuel source such as a fuel tank.

[0025] Figure 4A shows plunger **110**, the suction valve assembly, the check valve, and spring **114** in respective rest positions. While coil **116** is not energized, spring **114** biases, or urges, plunger **110** in direction **A2** such that the bumper spring is in contact with shoulder **136**. If backpressure exists, i.e., pressure caused by fluid entering from outlet port **106**, cap **142** forms a seal with seat **144** to prevent fluid from flowing from bore **112** past the suction valve assembly in direction **A1**. The seal in the check valve prevents fluid flowing from flowing past the check valve and out through the inlet port.

[0026] Figure 4B illustrates coil **116** as being energized, which forms a magnetic field. The magnetic field created by the energized coil imparts a directional force upon plunger **110** in direction **A1** toward inlet port **104**, causing the plunger to displace in direction **A1** and spring **114** to compress. As a result of the movement in direction **A1** and the configuration of the suction valve assembly, a negative pressure, or suction, is formed in chamber **158** of through-bore **108** and through-bore **112**, displacing cap **142** from seat **144**. Fluid present in chamber **156** in through-bore **108** just prior to energizing coil **116** is sucked around the suction valve assembly, as shown by flow lines **F1**, and into chamber **158** in through-bore **112**. During this stage, fluid is prevented from moving between chamber **156** and inlet port **102** by the check valve.

[0027] Referring now to Figure 4C, as coil **116** is de-energized, the magnetic field collapses. As a result, plunger **110** is no longer acted upon by a magnetic force and is urged in direction **A2** toward to the rest location of Figure 4A by the bias of spring **114**. Two simultaneous events occur during the movement of plunger **110** in direction **A2**. First, fluid contained in bore **112** and chamber **158** is forced out of outlet port **104**, as shown by fluid flow lines **F2**. The fluid in bore **112** and chamber **158** is prevented from entering chamber **156** by the seal created between cap **142** and seat **144**. Simultaneously, fluid is replenished in chamber **156** as follows. As plunger **110** moves in direction **A2**, a negative pressure, or suction, is created in chamber **156**. The negative pressure causes the check valve to open, allowing fluid to be drawn from inlet port **102** into chamber **156**, as shown by fluid flow lines **F3**.

[0028] The operation described above regarding Figures 4A through 4C is cyclically repeated during the use of the pump. As described below, the control unit energizes the solenoid coil for a particular time period $\mathbf{T}_{\text{off}}$, and de-energizes the solenoid coil for a particular time period $\mathbf{T}_{\text{on}}$ for example, while generating the power to operate the solenoid coil. This means that during each cycle of operation, the plunger is biased in direction **A1** by electromagnetic force for $\mathbf{T}_{\text{off}}$, and then biased in direction **A2** by spring **114** for $\mathbf{T}_{\text{on}}$. The reciprocal motion causes fluid to flow through inlet port **102** and the check valve into chamber **156**, through the suction valve assembly into chamber **158**, and through outlet port **106**, thereby creating a continuous flow of fluid.

[0029] As noted above, some amount of back pressure, that is, pressure exerted through the outlet port into through-bore **108** in direction **A1**, is typically present during operation of pump **100**. The back pressure biases the plunger in direction **A1**, against the biasing of spring **114**. When the force of the back pressure is greater than the force exerted by spring **114**, for example, spring **114** no longer can urge the plunger in direction **A2** from the position in Fig. 4B, the reciprocating action of the plunger is terminated and fluid no longer can be transferred as described above. Known solenoid pumps using nominal 12VDC input power cannot operate (pump fluid) above about 10 psi of back pressure.

[0030] Advantageously, pump **100** is able to operate (pump fluid) up to about 15 psi of back pressure. The ability of pump **100** to operate at greater back pressures is at least partly due to the variable rate of spring **114**. Due to the characteristics associated with operation of the solenoid coil, it is desirable to minimize the amount of resistance the plunger must overcome at the onset of a cycle. As noted above, the variable rate results in spring **114** advantageously generating relatively less biasing force resisting movement of the plunger in direction **A1** at the onset of a pump cycle, for example, starting in the position of Figure 4A. Also as noted above, the biasing force of spring **114** increases as the spring is compressed, such that in the position shown in Figure 4B, the biasing force is maximized. This maximized force initiates the movement of the plunger in direction **A2** after the coil is de-energized. Advantageously, the biasing force generated by spring **114** when the coil is de-energized determines the amount of back pressure under which pump **100** can operate. That is, the greatest amount of biasing force from spring **114** is needed to initiate displacement of the plunger against the back pressure when the solenoid coil is de-energized. Thus, spring **114** provides the least resistance when less resistance is advantageous, that is, when the solenoid coil is first energized and the displacement of the plunger in direction **A1** begins; and provides the most resistance when more resistance is advantageous, that is, when the solenoid coil is de-energized and spring **114** must operate against the back pressure.

[0031] Pump **100** can be used in common rail systems. As noted above, in a common rail system a relatively low pressure pump is used to pump fuel from a fuel source to a high pressure pump. For a common rail system, the back pressure on the outlet port of the low pressure pump is greater than the 10 psi maximum backpressure under which known solenoid pumps can operate. Advantageously, the approximately 15 psi maximum backpressure under which pump **100** can operate is sufficient to enable operation of pump **100** in a common rail system.

[0032] Figure 5A is a table showing duty cycle data for a solenoid pump using a control scheme varying a time for generating coil power **CP**.

[0033] Figure 5B is a table for a prior art control scheme with a fixed duty cycle. By duty cycle for a pump, we mean the percentage of the cycle during which the coil power is generated using the input voltage. Pump **100** is referenced in the discussion that follows; however, it should be understood that the control scheme described below is applicable to any solenoid pump using a solenoid coil to displace an element to transfer fluid from an inlet port for the pump to an outlet port for the pump. Control unit **118** is for controlling operation of the solenoid coil. The control unit is for accepting input voltage **IV**, for example, from an outside source, such as a battery of a vehicle in which the pump is installed. It should be understood that any source of voltage known in the art can be used to provide **IV**.

[0034] The control unit makes a determination regarding a magnitude of **IV** and generates **CP** during an interval equal to a time period $\mathbf{T_{off}}$. That is, the interval is the time period used by the control unit to generate **CP**. The control unit supplies the coil power to the solenoid coil. The control unit selects a duration of $\mathbf{T_{off}}$ such that the duration of $\mathbf{T_{off}}$ varies according to the determination of the magnitude of the input voltage. That is, the duration of $\mathbf{T_{off}}$ is proportional to the magnitude of **IV**. The combination of the magnitude of **IV** and the duration of $\mathbf{T_{off}}$ determine the magnitude of **CP** as further described *infra*.

[0035] The following should be viewed in light of Figures 4A through 5B. A cycle for pump **100** is defined as the time required for the pump to operate such that the plunger begins at the position shown in Figure 4A and returns to the position shown in Figure 4C. That is, a cycle is a cycle of operation for the plunger, spring **114**, and the pump to transfer a fluid from the inlet port to the outlet port. At the start of the cycle, the solenoid coil is de-energized by the control unit such that the plunger is in the position, within through-bore **108** and proximate the outlet port, shown in Figure 4A. To complete the cycle: the control unit energizes the solenoid coil by applying the coil power for time period $\mathbf{T_{off}}$ such that the plunger is displaced to the position, within sleeve **128** and proximate the inlet port, shown in Figure 4B; and the control unit de-energizes the solenoid coil by removing the coil power such that the plunger moves to the position in Figure 4C and then to the position shown in Figure 4A.

[0036] Advantageously, the control unit is for decreasing the duration of $\mathbf{T_{off}}$ as the magnitude of the input voltage increases; and increasing the duration of $\mathbf{T_{off}}$ as the magnitude of the input voltage decreases. In an example embodiment, the control unit compares **IV** to a pre-determined value. If **IV** is greater than the value, the control unit decreases $\mathbf{T_{off}}$ in proportion to the difference between **IV** and the value, with $\mathbf{T_{off}}$ decreasing as the difference increases. If **IV** is less than the value, the control unit increases $\mathbf{T_{off}}$ in proportion to the difference between **IV** and the value, with $\mathbf{T_{off}}$ increasing as the difference increases. Figure 5A shows an exemplary variation of $\mathbf{T_{off}}$ with respect the variation of **IV**. In an example embodiment, a minimum time period is necessary for the plunger to fully displace from the position shown in Figure 4A to the position shown in Figure 4B, and the control unit ensures that $\mathbf{T_{off}}$ is greater than the minimum time period.

[0037] As noted above, the control unit is for supplying the coil power to the solenoid coil during time period **T_{off}**. For an input voltage greater than a pre-determined value, the control unit is for selecting the duration of **T_{off}** to be less than the duration of **T_{on}**. For an input voltage less than the pre-determined value, the control unit is for selecting the duration of **T_{off}** to be greater than the duration of **T_{on}**. In an example embodiment, **T_{on}** is constant regardless of **T_{off}**.

[0038] As noted above, a duty cycle for a pump is defined as the percentage of the cycle during which the coil power is generated using the input voltage. For example, for a control scheme charging a capacitor with the input voltage to generate the coil power, the duty cycle is the percentage of the cycle during which the capacitor is charged. For the control scheme depicted in Figure 5A and described above, the duty cycle advantageously varies according to the magnitude of the input voltage. For example, in Figure 5A, the duty cycle decreases with increasing **IV**. In contrast, as shown in Figure 5B, the duty cycle is constant regardless of the value of **IV**, with attendant disadvantages and problems as described below.

[0039] In an example embodiment, **IV** is a direct current voltage and **CP** is an alternating current voltage. The control unit is for: supplying the coil power at a specific frequency; and selecting a magnitude of the frequency such that the magnitude of the frequency varies according to the magnitude of the input voltage. Thus, the control unit decreases the magnitude of the frequency as the magnitude of the input voltage decreases, and increases the magnitude of the frequency as the magnitude of the input voltage increases as shown in Figure 4A.

[0040] As shown in Figure 5B, and noted *supra*, known control schemes do not vary **T_{off}** or **CP** to account for changes in **IV**, that is, the duty cycle is constant. For example, in Figure 5B, **T_{off}** is 23 milliseconds (ms) regardless of the value for **IV**. As a result, a less than desirable amount of power is delivered to the solenoid coil for lower values of **IV**, for example, 10V in Figure 5B, resulting in incomplete displacement of the plunger by the solenoid and an undesirable decrease in pumping capacity for the pump. As the value of **IV** increases with the known control schemes, a different problem arises. At higher values of **IV**, for example, 14V in Figure 5B, the plunger is fully extended for a relatively long period before the expiration of **T_{off}**. As a result, the solenoid coil continues to be energized even though the plunger is fully extended, which leads to undesirable overheating of components in the pump, such as control circuitry. For example, electronic components in the circuitry, such as transistors, can overheat due to the preceding conditions. Further, the power efficiency of the pump is decreased since excessive amounts of power are consumed by components in the pump, such as the control circuitry, without producing any additional useful work.

[0041] Figure 6 depicts exemplary power circuit **220** for a control scheme varying a time for generating coil power according to input voltage. The following should be viewed in light of Figures 4A through 6. Pump **100** is used as an example in the discussion that follows. However, it should be understood that the control scheme described below is applicable to any pump using a solenoid coil to displace an element to transfer fluid from an inlet port for the pump to an outlet port for the pump and is not limited to pump **100**. In an example embodiment, control unit **118** includes circuit **220** shown in Figure 6. Although circuit **220** is described with respect to control unit **118**, it should be understood that circuit **220** is applicable to any pump using a solenoid coil to displace an element to transfer fluid from an inlet port for the pump to an outlet port for the pump and is not limited to control unit **118**.

[0042] In an example embodiment, control unit **118** includes power input line **222**, power circuit **220** includes voltage storage element **C2**, and the control unit is for charging the voltage storage element with the input voltage to generate the coil power during the interval noted above for T_{off}, and discharging the voltage storage element to supply the coil power to the solenoid coil. In an example embodiment, element **C2** is a capacitor.

[0043] In an example embodiment, circuit **220** includes transistor **Q1**, for example, a metal oxide semiconductor field effect transistor (MOSFET), and timer **U1**. Timer **U1** can be any timer known in the art, for example, a 555 timer. In an example embodiment, pin **5** on the timer is clamped to establish a predetermined value against which the input voltage is compared. Pin **5** is the control voltage for a comparator circuit in the timer. In an example embodiment, a Zener diode, for example, diode **D6** is used to clamp pin **5**. To produce the values shown in Figure 5A, the voltage is clamped at 5.1V; however, it should be understood that other clamping voltage values are possible. The timer turns **Q1** off during T_{on} such that the coil is de-energized and **C2** is charged. The timer turns **Q1** on during T_{off} such that **C2** is discharged and the coil is energized.

[0044] The control scheme described above, for example, selecting the duration of T_{off} according to a magnitude of **IV**, has at least the following advantages. In many applications, the magnitude of **IV** varies according to operating conditions affecting the source of **IV**. For example, when the pump is used in a vehicular application and a battery for a vehicle is used to supply **IV**, the magnitude of **IV** may be relatively lower due to the age or condition of the battery, cold weather impacting the battery, or a start-up condition for the vehicle. As a result, the magnitude of **IV** may be undesirably low at the onset of operation of the pump and may increase as the vehicle continues to operate, for example, as the battery warms up or is charged.

[0045] Thus, during typical operation, it is expected that **IV** will vary, for example, as shown in Figures 5A and 5B. As noted *supra*, known control schemes do not vary the duty cycle to account for such variations of **IV**. Thus, undesirably low power is delivered to the solenoid for lower input voltage values, resulting in a loss of pumping performance, and excessive power is delivered to the solenoid for larger input voltage values, resulting in overheating of components in the pump and excessive power consumption by the pump.

[0046] Advantageously, the control scheme described *supra* for Figures 5A and 6 matches generation of **CP** to actual **IV** conditions, for example, controlling a duty cycle according to actual **IV** conditions. As a result, **CP** is increased at lower levels for **IV** to ensure optimal pumping rates, and **CP** is reduced at higher levels to avoid overheating components and to increase energy efficiency.

[0047] Thus, it is seen that the objects of the invention are efficiently obtained, although changes and modifications to the invention should be readily apparent to those having ordinary skill in the art, without departing from the spirit or scope of the invention as claimed. Although the invention is described by reference to a specific preferred embodiment, it is clear that variations can be made without departing from the scope or spirit of the invention as claimed.

What is claimed is:

1. A solenoid pump, comprising:

an inlet port, an outlet port, and a first through-bore connecting the inlet and outlet ports;

a plunger disposed within the first through-bore and including a second through-bore;

a spring arranged to urge the plunger toward the outlet port;

a solenoid coil disposed about a portion of the plunger and arranged to displace the plunger toward the inlet port in response to direct current coil power applied to the solenoid coil; and,

a control unit including:

an input for accepting a direct current input voltage; and,

a power circuit for:

receiving the direct current input voltage;

generating the direct current coil power during an interval equal to a first time period;

supplying the direct current coil power to the solenoid coil; and,

selecting a duration of the first time period such that the duration of the first time period varies according to the direct current input voltage, wherein the control unit is for:

comparing the direct current input voltage to a pre-determined value; and,

selecting the duration of the first time period according to a difference between the direct current input voltage and the pre-determined value.

2. The solenoid pump of claim 1, wherein the control unit is for:

decreasing the duration of the first time period as a magnitude of the direct current input voltage increases; and,

increasing the duration of the first time period as the magnitude of the direct current input voltage decreases.

3. The solenoid pump of claim 1 or 2, wherein:

the control unit includes a voltage storage element; and,

the control unit is for:

charging the voltage storage element with the direct current input voltage to generate the direct current coil power during the interval; and,

discharging the voltage storage element to supply the direct current coil power to the solenoid coil.

4. The solenoid pump of any one of claims 1 to 3, wherein the control unit is for:

supplying the direct current coil power at a frequency; and,

selecting a magnitude of the frequency such that the magnitude of the frequency varies according to the magnitude of the direct current input voltage.

5. The solenoid pump of any one of claims 1 to 4, wherein the control unit is for:

decreasing the magnitude of the frequency as the magnitude of the direct current input voltage decreases; and,

increasing the magnitude of the frequency as the magnitude of the direct current input voltage increases.

6. The solenoid pump of any one of claims 1 to 5, wherein:

the power circuit includes a timer; and,

the control unit is for using the timer to set the pre-determined value.

7. A solenoid pump, comprising:

an inlet port, an outlet port, and a first through-bore connecting the inlet and outlet ports;

a plunger disposed within the first through-bore and including a second through-bore;

a spring arranged to urge the plunger toward the outlet port;

a solenoid coil disposed about a portion of the plunger and arranged to displace the plunger toward the inlet port in response to direct current coil power applied to the solenoid coil; and,

a control unit including:

an input for accepting a direct current input voltage;

a power circuit:

for:

receiving the direct current input voltage;

generating direct current coil power during an interval equal to a first time period; and,

supplying the direct current coil power to the solenoid coil; and,

including a timer configured to select a duration of the first time period such that the duration of the first time period varies according to the direct current input voltage.

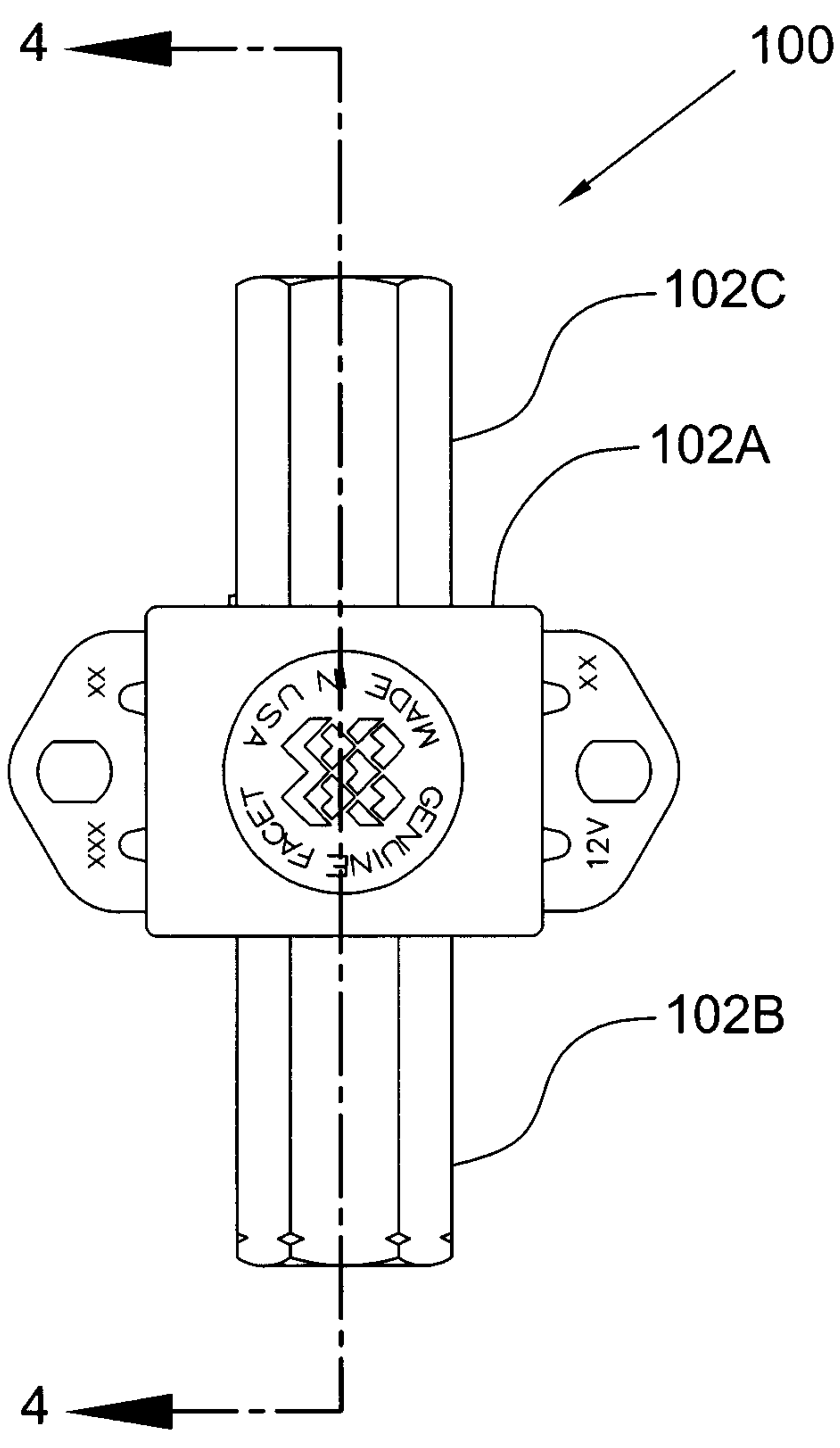

4
100
102C
102A
XX
XX
XXX
12V
GENUINE FACET
MADE IN USA
102B
4

Fig. 1

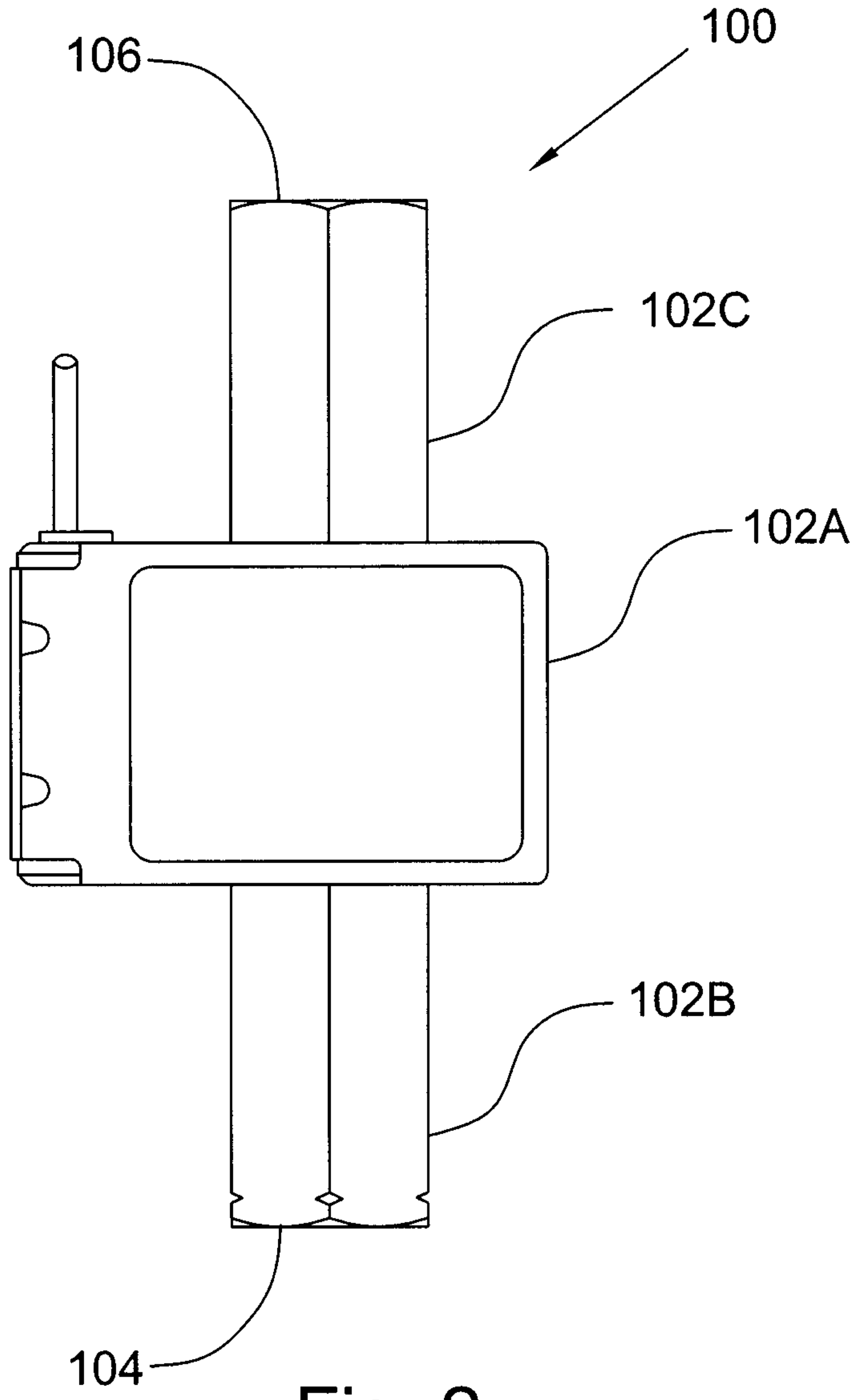

106
100
102C
102A
102B
104

Fig. 2

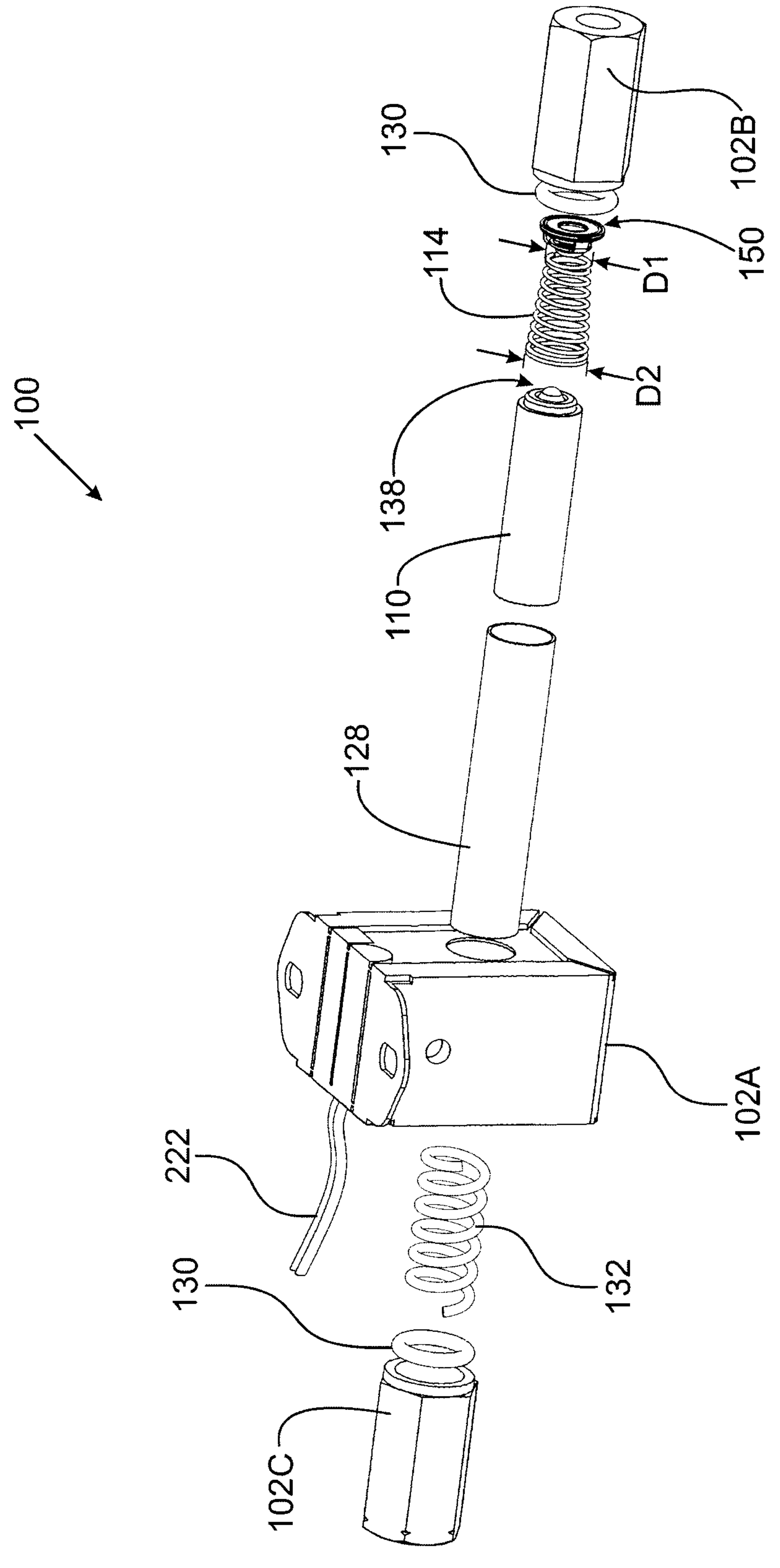


Fig. 3

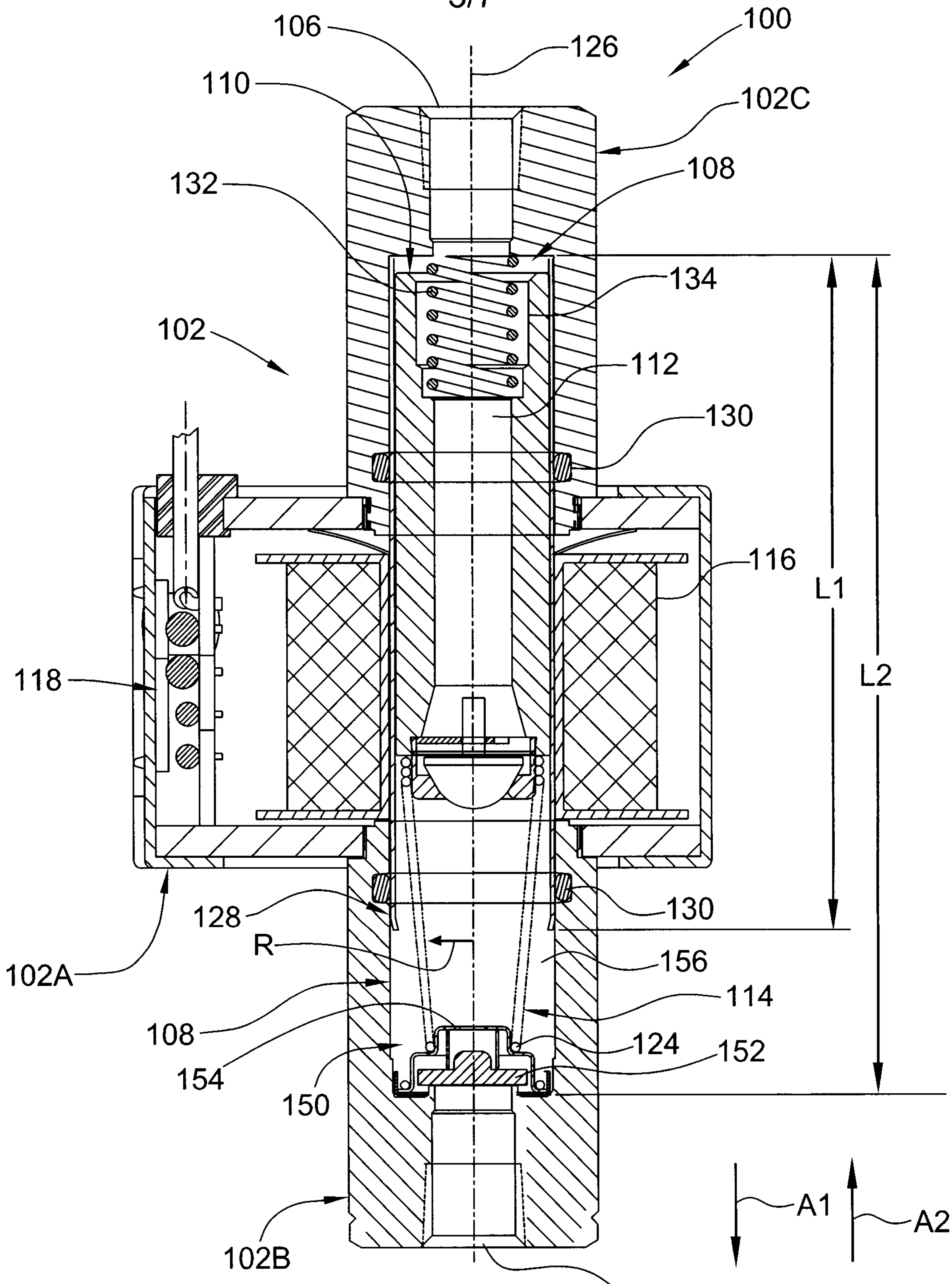
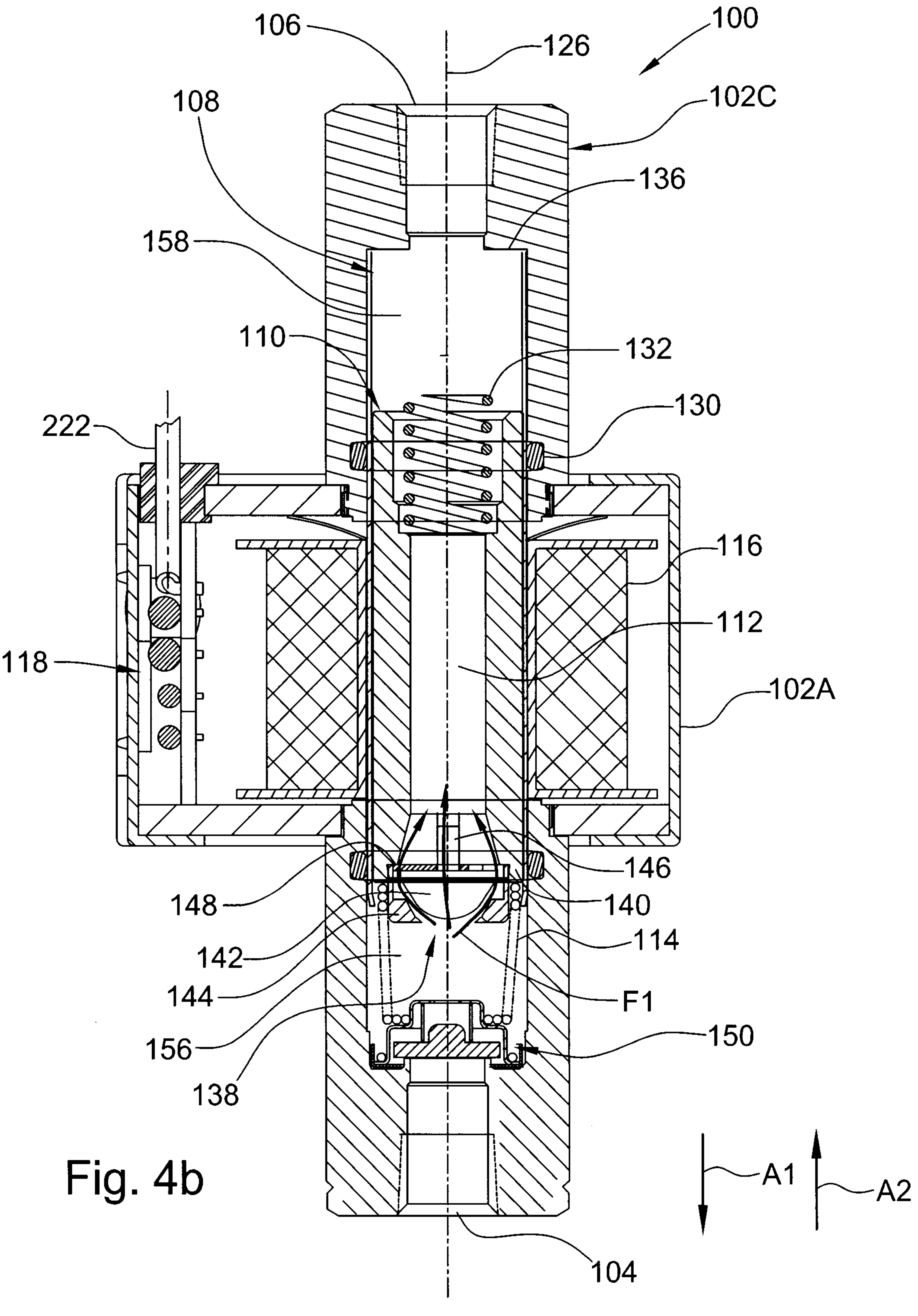

106
126
100
110
102C
108
132
134
102
112
130
116
L1
L2
118
130
128
R
102A
156
114
108
124
154
152
150
A1
A2
102B
104

Fig. 4a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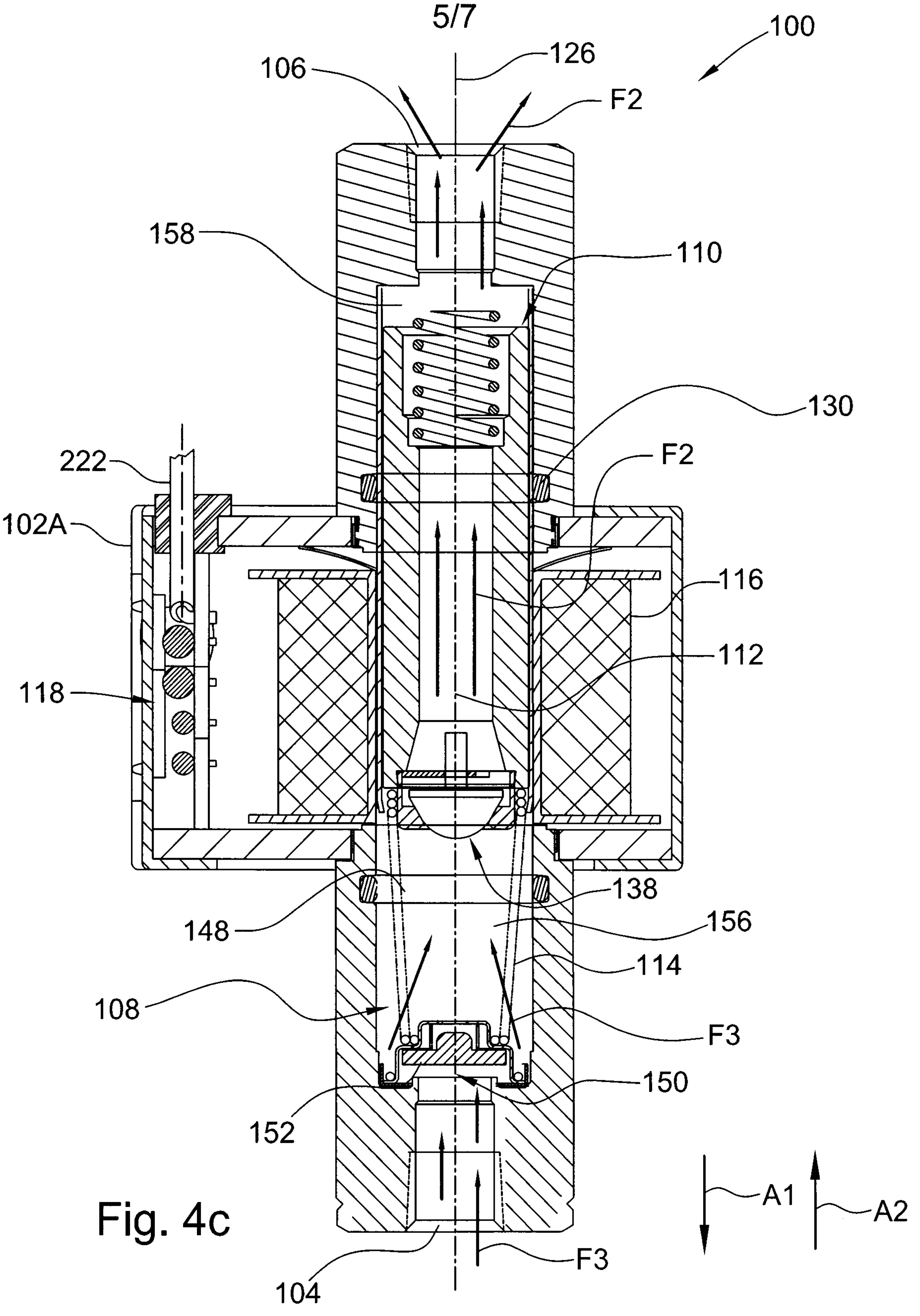
106
126
102C
108
136
158
110
132
130
222
116
112
118
102A
146
148
140
142
114
144
F1
156
150
138
A1
A2
104

Fig. 4b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106
126
F2
158
110
130
222
F2
102A
116
112
118
138
148
156
114
108
F3
150
152
104
F3
A1
A2

Fig. 4c

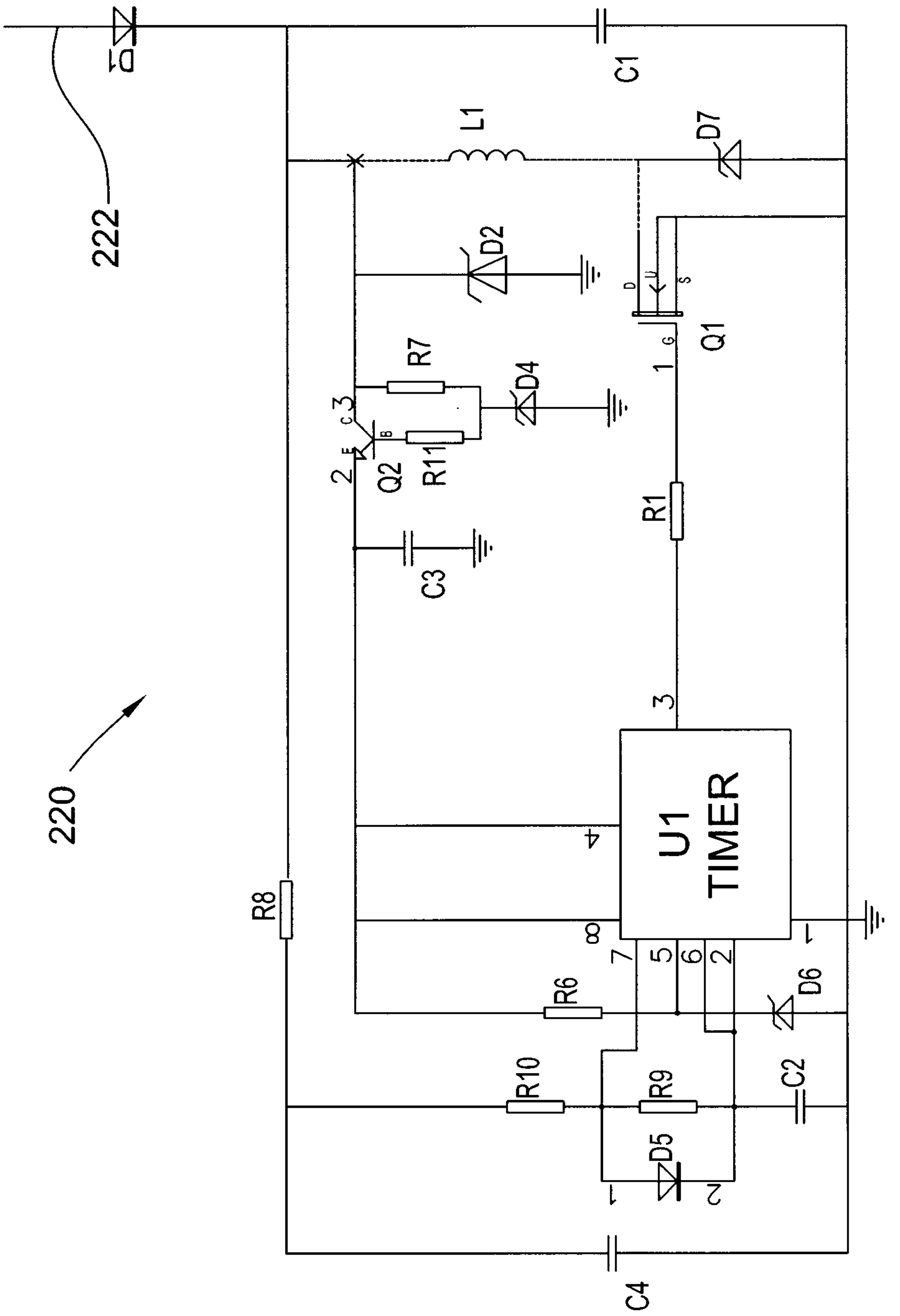
Modified Astable Circuit							
Vdc	Toff (ms)	Ton (ms)	Duty Cycle	Frequency (Hz)	Average Current (A)	Max Current	Average Power
10.00	30.00	23.00	56.60	18.87	1.70	3.00	16.98
12.00	21.00	23.00	47.73	22.73	2.39	5.00	28.64
14.00	16.00	23.00	41.03	25.64	2.26	5.50	31.59

Fig. 5a

Astable Circuit							
Vdc	Toff (ms)	Ton (ms)	Duty Cycle	Frequency (Hz)	Average Current (A)	Max Current	Average Power
10.00	23.00	23.00	50.00	21.74	1.50	3.00	15.00
12.00	23.00	23.00	50.00	21.74	2.50	5.00	30.00
14.00	23.00	23.00	50.00	21.74	2.26	5.50	38.50

PRIOR ART

Fig. 5b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220
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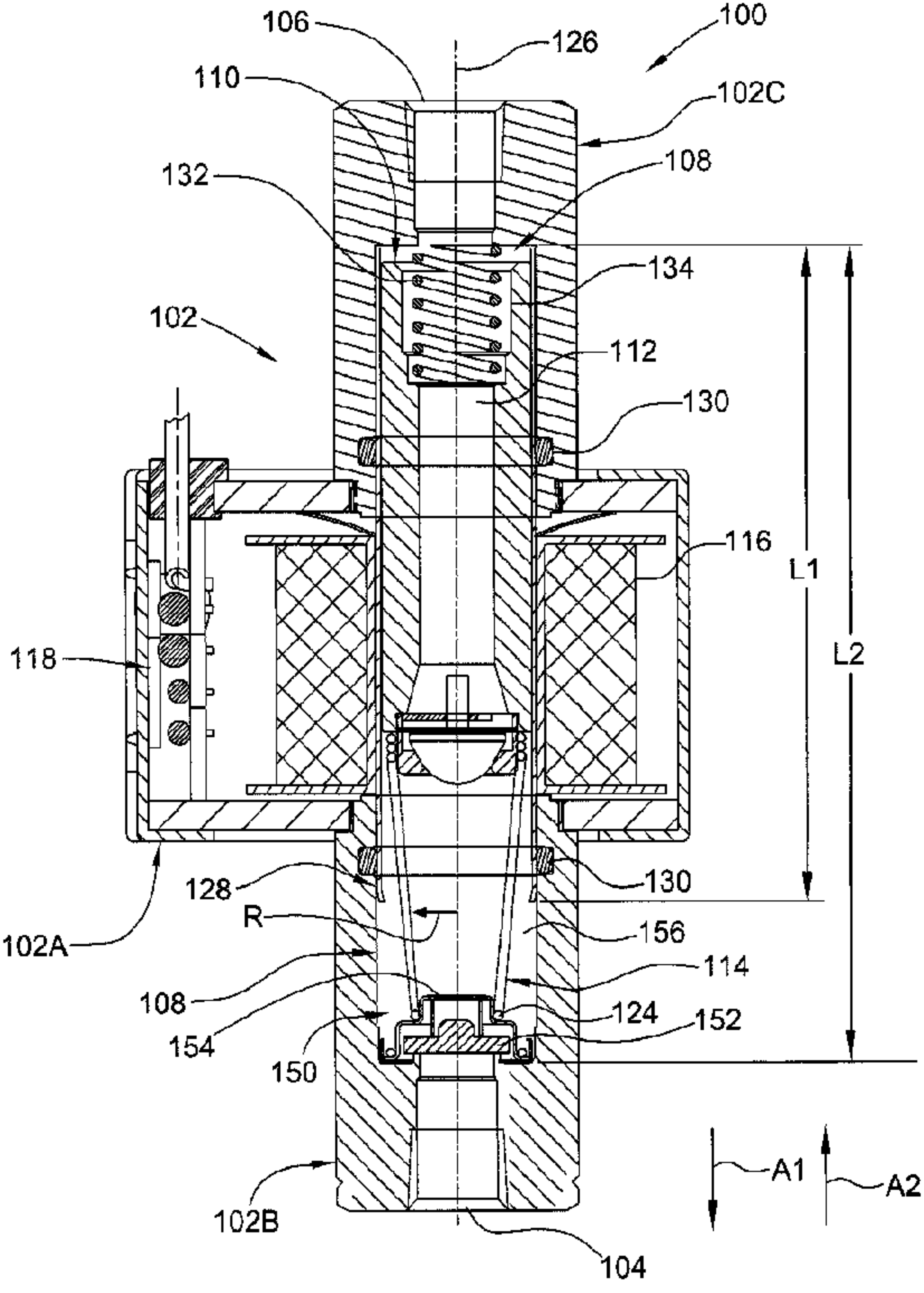
C1
L1
D7
D2
D
U
S
G
Q1
1
R7
D4
C
3
B
E
2
Q2
R11
R1
C3
3
U1
TIMER
4
8
7
5
6
2
1
R8
R6
D6
R10
R9
C2
D5
1
2
C4

Fig. 6


106
100
126
110
102C
108
132
134
102
112
130
116
L1
L2
118
128
130
R
156
102A
114
108
124
152
154
150
A1
A2
102B
104