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(54) **Machine gun.**

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Description

This invention relates to machine guns having two gun barrels and a linkage which alternatively drives each breech bolt via a cam operated accelerator.

Two barreled machine guns having the recoil of the exploded charge in one barrel loading and locking the charge in the other barrel have been shown in the following U.S. patents:

390,114, issued Sept. 29, 1888, to B. Burton;
623,003, issued April 11, 1899, to F. M. Garland;
636,974, issued Nov. 14, 1899, to F. M. Garland;
1,332,060, issued Feb. 24, 1920, to L. Paeilli;
1,351,017, issued Aug. 31, 1920, to C. C. Blackmore;
1,477,115, issued Dec. 11, 1923, to C. Gast;
1,856,022, issued April 26, 1932, to L. V. S. Blacker; and
2,972,286, issued Feb. 21, 1961, to F. R. Marquardt.

DE—A1—3208942 describes a machine gun in accordance with the preamble of claim 1 and, thus, including an internal drive means comprising a motor coupled to a lever.

Handbook "Waffentechnisches Taschenbuch", Rheinmetall, Dusseldorf, 1980 page 257 describes a machine gun with external drive means which is either an electrical, a hydraulic or pneumatical means.

It is an object of this invention to provide a lightweight, high rate of fire, gun mechanism for a two barrel gun which is driven by an external drive.

Another object is to provide a breech bolt, bolt accelerator and locking mechanism for a two barrel gun having a rocking lever to synchronize the sets of mechanisms.

Yet another object is to provide a mechanism for two barrel gun wherein a linear motion input from the external drive imparts an oscillatory motion to the rocking lever.

According to the present invention there is provided a machine gun comprising:

1. a receiver (12; 26)
- a pair of gun barrels (10L; 10R) fixed to said receiver (12; 26)
- a pair of operating mechanisms, and a pair of gun bolt assemblies (118 L/R) including a pair of gun bolts, disposed in said receiver (12, 26) for loading, locking and firing each of said gun barrels (10/R) respectively;
- a rocking lever (34) coupled to and between said operating mechanisms for driving said mechanisms 180° out of mutual phase;
- said rocking lever (34) journaled for pivotal movement about a central axis (36); and
- external drive means comprising a motor (460) coupled to said rocking lever (34) for reciprocating said lever about said axis (36) characterized in that
- said external drive means includes:
 - a pinion gear (468) driven by the motor (460);
 - said pinion gear (468) meshed with and driving two mutually opposed gears (466, 470), respec-

tively journaled for rotation about a common axis (460), in opposite directions of rotation;

a cam follower (482) having a cleft cam surface (484) disposed between said two opposed gears (466, 470) and journaled for oscillation about an axis (480) to and between two angularly spaced apart dispositions;

two cams (472, 476) each carried by a respective one of said two opposed gears (466, 470); and one of said cams (472) adapted to enter said cleft (484) at one of said dispositions and to swing said follower (482) to the other of said dispositions and to exit said cleft, and the other of said cams (476) adapted to enter said cleft at said other of said dispositions and to swing said follower to said one of said dispositions and to exit said cleft.

These and other objects, features and advantages of the invention will become apparent from the following specification thereof taken in conjunction with the accompanying drawing in which:

FIG. 1 is a perspective view of a gun embodying this invention;

FIG. 2 is a detail of FIG. 1 showing the breech and receiver assembly;

FIG. 3 is a detail of FIG. 2 showing the feeder mechanism;

FIG. 3A is a detail of FIG. 3 in cross-section along a vertical, longitudinal plane showing the feeder shafts;

FIG. 4 is a detail of FIG. 2 showing the operating linkages;

FIG. 5 is a side view in elevation of the left hand slider, the left hand aft stripper and the left hand forward stripper;

FIG. 6 is a detail of FIG. 2 showing the gun bolts and their linkages;

FIG. 6A is a detail of FIG. 6 showing various guide surfaces on the breech housing;

FIG. 7 is a detail of FIG. 6 showing the quick release connection between the gun bolt rod and the accelerator rod;

FIG. 8 is a detail side view in elevation of the bolt head with the firing pin in its retracted disposition;

FIG. 9 is a view similar to FIG. 8 but with the firing pin in its fired disposition;

FIG. 10 is a detail perspective view showing the interaction of the strippers with the belt of linked ammunition, here showing the left hand strippers about to strip a round for the left hand gun barrel while the left hand gun bolt holds an extracted cartridge case;

FIG. 11 is a detail view showing the left hand gun bolt locked by the left hand forward stripper as it holds a round in the chamber while the extracted cartridge case has been ejected;

FIG. 12 is a perspective view of the left hand, center, and right hand guide for the rounds as disposed in the breech housing as shown in FIG. 2;

FIG. 12A is a detail cross-section of the center guide of FIG. 12;

FIGS. 13 and 14 are top and side elevation

views respectively of the left hand and center guides, the left hand aft and forward strippers, and the left hand gun bolt cooperating with a round still in its link in the feed disposition;

FIGS. 15 and 16 are similar to FIGS. 13 and 14 but show the round stripped from its link;

FIGS. 17 and 18 are similar to FIGS. 13 and 14 but show the round carried forward towards the chamber;

FIGS. 19 and 20 are similar to FIGS. 13 and 14 but show the round chambered and the bolt locked;

FIGS. 21 and 22 are similar to FIGS. 13 and 14 but show the fired case extracted and the bolt half-way towards aft dwell;

FIGS. 23 and 24 are similar to FIGS. 13 and 14 but show the fired case extracted and the bolt seven-eighths towards aft dwell;

FIGS. 25 and 26 are similar to FIGS. 13 and 14 but show the fired case extracted and the bolt in aft dwell;

FIGS. 27 and 28 are similar to FIGS. 13 and 14 but show the fired case half-way ejected, leading to the fully ejected disposition shown in FIG. 11;

FIGS. 29 and 30 are a side view in elevation and an aft view looking forward in elevation respectively of the hammer assembly shown in the on-sear or safed disposition;

FIGS. 31 and 32 are similar to FIGS. 29 and 30 respectively, but show the hammer assembly in the off-sear or firing disposition;

FIGS. 33 through 37 are detail aft views of the hammer, the left and right hand forward strippers and the respective tail portions of the left and right hand firing pins, showing their cooperation during various stages of the gun operating cycle;

FIGS. 38 and 39 are a perspective view and a detail side view in elevation respectively of the gas drive and clearing assembly, showing the assembly in its deenergized or cleared disposition;

FIGS. 40 and 41 are similar to FIGS. 38 and 39 respectively, but showing the assembly in its energized or firing disposition;

FIGS. 42 and 43 are a top view and a side view respectively in elevation of an external drive, and having a rotary input coupled to the operating mechanism; and

FIG. 44 is a perspective view, similar to FIG. 4, showing an external drive having a rectilinear, reciprocating input coupled to the operating mechanism.

As seen in FIG. 1, the gun includes a left hand gun barrel 10L and a right hand gun barrel 10R which are secured to a breech housing 12 of a breech assembly 14. The barrels are also constrained by a muzzle clamp assembly 16, a mid-barrel clamp assembly 18, and an aft clamp assembly which is part of a gas drive assembly 22 and is shown in FIG. 1, and in FIGS. 38 through 41, for driving the gun, but this gas drive assembly may be deleted and the externally powered drive assembly shown in FIGS. 42 and 43 or the externally powered drive assembly shown in FIG. 44 may be substituted therefor. The breech hous-

ing 12 is fixed to a receiver housing 26 in a receiver assembly 28. A hammer assembly 30 is also fixed to the breech housing 12. The breech and receiver housings may be considered together to be the classical "receiver".

As seen in FIG. 4, the operating mechanism 32 is mainly disposed within the receiver housing 26 and is included in the receiver assembly 28. This operating mechanism includes a rocking lever 34 which is fixed to a central post 36, and which post is journaled for oscillation to the receiver housing 26. The rocking lever 34 has a left arm 34L to which is fixed a left post 38L which is journaled to the forward end of a left link 40L, and has a right arm (not visible) to which is fixed a right post (not visible) which is journaled to the forward end of a right link 40R. The aft end of the left link 40L is journaled to a left post 42L which is fixed to a left slider 44L. The aft end of the right link 40R is journaled to a right post (not visible) which is fixed to a right slider 44R which is identical to the left slider 44L. The aft end of each link 40L/R has a projection 40P which is captured in a respective notch in the respective slider 44L/R to capture the link to its respective post 42L/R. An additional post 4E is fixed to the rocking lever 34 and is journaled to the forward end of a feeder arm 48. The aft end of the feeder arm 48 has a post (not visible) to which is pivotally journaled a tail slider 52. The aft end of the slider has a cross-piece 54 which rides in a channel 56 in the receiver housing 26. The forward end of the slider 52 has a cam driver 58 with two mutually spaced apart upstanding arms which terminate in mutually opposed cam driving elements 60L and 60R. The cam driver 58 also has a pair of slots 62L and 62R which ride on a pair of tracks (not visible) in the receiver housing. The cam driver 58 is thereby constrained to rectilinear reciprocating movement when driven by the feeder arm 45 whose forward end is free to oscillate laterally while reciprocated fore and aft by the rocking lever 34.

The operating mechanism also includes a left forward stripper 64L, a right forward stripper 64R, a left aft stripper 66L and a right aft stripper 66R. The forward strippers are disposed in respective vertical passageways 68L and 68R in the breech housing 12 and 70L and 70R in the receiver housing 26. The aft strippers are disposed in respective vertical passageways (not visible) in the receiver housing 26. Each forward stripper includes a base portion 74 and two upstanding arms 76L and 76R each having a respective, inwardly directed lug 78L and 78R, and a hammer upper camming surface 79U and a hammer lower camming surface 79D. Each base 74 of each forward stripper has a respective diagonally downwardly facing cam 80C and a part of diagonally-upwardly facing cam surfaces 82L/R. Each aft stripper 66 has a respective upwardly directed lip 84, a diagonally-downwardly facing cam surface 86C and a pair of diagonally-upwardly facing cam surfaces 88L/R.

Each of the sliders 44L/R has a transverse slot 90 in its forward end which receives a disk 92L/R

on the end of a respective piston rod 94L/R in the gas drive assembly 22. As each piston rod is reciprocated by the gas drive assembly it reciprocates the thereto engaged slider. The two sliders, by means of the respective links 40L/R, oscillate the rocking lever 34 about its central post 36.

Each of the sliders 44L/R has a respective diagonally-upwardly facing cam surface 96C adapted to drive the stripper cam surface 80C upwardly to bring the respective forward stripper up to engage the nose of a round which is to be stripped from its link, and has a diagonally-upwardly facing cam surface 98C adapted to drive the stripper cam surface 86C to bring the respective aft stripper up to engage the base of a round which is to be stripped from its link.

Each of the sliders 44L/R has a respective pair of diagonally-downwardly facing cam surfaces 100L/R adapted to drive the respective pair of forward stripper cam surfaces 82L/R downwardly to bring the respective stripper downwardly, and has a respective pair of diagonally-downwardly facing cam surfaces 102L/R adapted to drive the respective pair of stripper cam surfaces 88L/R downwardly to bring the respective aft cam stripper downwardly, the motion of the strippers being required before the breech bolt begins its ram stroke.

Each of the sliders 44L/R has a respective pair of diagonally-downwardly directed cam surface 104L/R to drive the respective pair of stripper cam surfaces 82L/R further downwardly to bring the respective forward stripper further down into its gun bolt locking disposition, and has a respective pair of downwardly facing cam surfaces 106L/R to over-ride a respective pair of upwardly facing cam surfaces 108L/R on the forward stripper to maintain such forward stripper in its gun bolt locking disposition.

Each of the sliders 44L/R has a respective cam surface 108C to drive the respective stripper cam surface 80C upwardly to bring the respective forward stripper out of its gun bolt locking disposition.

Each of the sliders 44L/R has a respective upstanding ear portion 110 with a cross bore 112 therethrough, and has a forward pair of outwardly-laterally-extending tracks 114L/R and an aft pair of outwardly-laterally-extending tracks 116L/R. These tracks ride in fore-aft extending respective tracks (not visible) in the receiver housing 26.

Each of a pair of gun bolt assemblies 118L/R respectively includes a bolt head assembly 120L/R, a connecting link 122L/R and an accelerator link 124L/R. Each bolt head assembly 120L/R includes a bolt head body 126L/R having an interior cavity 128 with an aftwardly directed opening 130 to the body aft face 132, an upwardly directed opening 134 to the body top face 136, and a forwardly directing opening 138 to the body front face 140. Each body 126L/R also has a pair of upper extractor lugs 142L/R and a pair of lower extractor lugs 144L/R which are adapted to enter the

extractor groove 146 forward of the base 148 of the cartridge cas 150 of rounds 152 of ammunition. Each body also has a pair of mutually spaced apart, upwardly directed ears 154L/R with a respective cross-bore 156L/R. A firing pin 158 is mounted in each bolt head and includes an upper head 160 over lying the bolt top face and extending aftwardly from an elbow 162 with a cross-bore 164 which is pivotally secured between the ears 154L/R by a cross-pin 166 through the bores 156L/R and 164, a leg 168 extending downwardly from said elbow through the opening 134 into the cavity 128 and terminating in a penetrator 170 extending through the opening 138. When the base 148 of a cartridge case 150 is held against the front bolt face 140 it presses against the tip of the penetrator 170 to pivot the distal end of the head 160 upwardly. When the head 160 is struck downwardly by the hammer assembly 30 it pivots the penetrator forwardly into the primer cup in the base 148 of the cartridge base to fire the round of ammunition. After firing, as the bolt head moves upwardly in unlocking, the base of the case moves relatively downwardly on the bolt face, and the primer cup presses on the upper surface of the tip to pivot the tip aftwardly into the bolt head. The bolt head body 126 also has a pair of laterally extending channels 172L/R adapted to receive respective tracks (not visible) in the breech housing 12 and the lugs 78L/R respectively in the respective forward stripper 64L/R. The four aft surfaces 173a, b, c and d serve as locking lugs for the bolt head body 126 and abut adjacent surfaces 174a, b, c and d (174b, c and d not visible) on the breech housing which serve as locking lugs for the breech. The bolt head body is shifted vertically into lock and unlock disposition by the respective forward stripper through the engagement of forward stripper lugs 78L/R with the bolt head body channels 172L/R.

Each connecting link 122L/R has a forward end with a cross-bore 175 disposed within the cavity 128 and which is pivotally held to the bolt body 126 by a cross-pin 176 disposed in the cross-bore 175 and in two aligned bores (not visible) in the bolt body 126. The link has an aft end with a quick release assembly 178 shown in FIG. 7.

Each accelerator link 124L/R has a lower portion terminating in a forwardly extending clevis 180 having a central slot 182 defined by the two ears of the clevis 180 and having a cross-bore 184L/R through each of the ears respectively. The aft ear 110L/R of the respective slider 44L/R is disposed within the slot 182 and is pivotally held to the clevis by a cross-pin 186 through the cross-bore 112 and the cross-bores 184L/R. The lower portion also has a cross-bore (not visible) through which passes a shaft 190 which has an outer hand cam roller 192 integral therewith and an inner hand cam roller 194 journaled thereon. These cam rollers ride on respective S-shaped cam surfaces 196L/R in the receiver housing 26 which has upper and lower outer-upstanding lips 198 and 200 to preclude the assembly of cam rollers 192 and 194 from leaving the cam surfaces. Respective

notches 202 and 204 are cut in the lips to permit the transverse entry of the rollers when the link is assembled into the housing.

Each accelerator link 124L/R has an upper portion terminating in a upwardly extending clevis 206 having a central slot 208 defined by the two ears of the clevis and having cross-bores 210L/R through each of ears respectively. A cross-pin 212 is disposed through the cross-bores 210L/R across the slot 208 and has a cross-notch 214 of semi-circular transverse cross-section. The quick release assembly 178 interengages the clevis 206 and cross-pin 212 to provide a pivotable joint between the accelerator link 124 and the connecting link 122. The quick release assembly 178 includes a cross-bore 216 through the aft end portion of the connecting link and a notch cutting away a corner of the end portion to the cross-bore 216 to provide an aft face 218 which is tangential to the circumference of the cross-bore 216 and an upper face 220 which is perpendicular to the aft face 218 and has an opening equal to the diameter of the pin. These two faces and the remaining circumferential wall of the cross-bore will pass the cross-pin 212 when the end portion of the connecting link is moved into the proximal portion of the slot 208 and will nest the cross-pin 212 into the cross-bore 216 when the end portion is moved towards the distal portion of the slot 208. The end portion has a longitudinal blind bore 222 in which are disposed a helical compression spring 224 and a plunger-rod 226. The plunger-rod has a longitudinally extending flat 228 which passes a cross-pin 230 which is fixed in a transverse bore in the aft end portion of the connecting link through the longitudinal blind bore 222. The compression spring 224 biases the plunger-rod 226 aftwardly, within the constraint provided by the flat 228 and the cross-pin 230, so that the distal end of the plunger-rod 226 is disposed within the notch 214 of the cross-pin 212 to capture the cross-pin 212 to the quick release assembly 178. To disconnect the connecting link 122 from its respective accelerator link 124, the plunger-rod 226 is pressed, against the bias of the spring 224, out of the notch 214 and the quick release assembly is unhooked from the cross-pin 212.

The accelerator link 124, in conjunction with the S-shaped cam surface 196, converts the motion of the slider 44 provided by the gun drive mechanism into the motion required for the operating cycle of the gun bolts.

An ammunition feeder assembly 234 is disposed within the receiver housing 26 and includes a transverse guide 236 having a forward groove 238 and an aft groove 240 to guide the forward and aft ends of links 241 which form the ammunition belt, here shown as a left hand feed. The links are of the side stripping type wherein each round of ammunition provides a pivotal interconnection between immediately adjacent links. A four toothed sprocket 242 is fixed to a longitudinally extending forward feeder shaft 244 whose forward and aft ends are journaled for rotation in the

receiver housing. A drum cam 246, having four terminally interconnected helical grooves 248, is splined to an aft shaft 245. The aft end of the shaft 244 is slotted to slidably receive the blade forward end of the aft shaft 245 whose aft end is journaled in the longitudinal bore of a one-way clutch 247 which serves as its aft end journal in the receiver housing. A helical compression spring 245a is captured between a plunger 245b held by a crosspin 245c to bias the aft shaft blade into the slot. The blade may be manually withdrawn to permit reverse rotation of the forward shaft to permit the ammunition belt to be manually withdrawn. The cam driving elements 60L and 60R of the cam driver 58 ride the grooves 248. When the feeder arm 48 is driven aft by the rocking lever 34 the cam driver 58 drives the drum cam, and the shafted thereto sprocket 242 through 90° of clockwise rotation, and when the feeder arm is driven forward the cam driver drives the drum cam and the sprocket through an additional 90° of clockwise rotation. Each full cycle of the rocking lever thus advances the ammunition belt by two rounds of ammunition. Considering the leading four rounds in the ammunition belt during left side feed operation, it will be seen that on the aft stroke of the feeder arm the sprocket aligns the second round with the left hand gun bolt and gun barrel and strips that second round from its link, and on the forward stroke of the feeder arm this second round is chambered into the left hand gun barrel, while the sprocket aligns the first round with the right hand gun bolt and gun barrel and strips that first round from its link. On the following aft stroke of the feeder arm this first round is chambered into the right hand gun barrel, while the sprocket aligns the fourth round with the left hand gun bolt and strips it from its link. On the following forward stroke of the feeder arm this fourth round is chambered into the left hand gun barrel, while the sprocket aligns the third round with the right hand gun bolt and strips it from its link. The above described procedure can be done in a right side feed operation by reversing the sprocket, the drum cam, and the one way clutch. The tips of the teeth of the sprocket are spaced apart by a pitch which is slightly greater than the normal, unstressed pitch of the belt of ammunition to accommodate any stretching of the belt.

The stripping of each round is performed by the respective set of forward and aft strippers while the ramming of such round is performed by the respective gun bolt and an array of guides in the breech assembly 14. The guide array includes a center guide 250, a left guide 252L and a right guide 252R as well as the ammunition link and guides machined in the breech housing 12. As seen in FIGS. 12 through 28, the center guide 250 is pivotally mounted on a pin 253 to a longitudinally extending web 254 in the breech housing 12. The guide has a downwardly directed pocket 256 formed therein with an outwardly extending overhanging lip 258. A detent plate 2360 is disposed in the pocket and has a forwardly extending foot 262 under the lip 258 and a helical

compression spring 264 captured in an upwardly directed pocket 266. The spring biases the detent plate upwardly about the transverse pivot axis formed by the foot and the lip. The detent plate also has a upwardly extending nose 268 adapted to enter either of two slots (not visible) in the breech housing 12 to retain the center guide in either of its swung left or swung right dispositions, and a left and right cam surface 272L/R which may be overridden by an aftwardly moving bolt head to pivot the detent plate downwardly to disengage its nose 268 from one of said slots. The center guide also has a pocket 274C which receives a downwardly extending projection (not visible) in the breech housing 12 to limit the swing to the left or right of the center guide; and has a left and a right cam surface 276L/R adapted to be abutted by the aftwardly moving bolt head, after the bolt head has released the detent, whereby the bolt head can swing the center guide to the right or left respectively. The center guide also has a left and a right concave upper guide surface 278L/R and a left and a right concave lateral guide surface 180L/R.

The left and right side guides 252L/R each include a forward side guide 282 and an aft side guide 284 which are both pivotally journaled on a pin 286 to the breech housing 12. A compression spring 228 is captured in a concave groove in a slot in an aftwardly extending leg 294 of the forward side guide 282 and biases a forwardly extending foot 296 of the aft side guide 284 which extends into the slot. The aft tips of the leg 294 are thus biased against upper and lower stops 298U/L on the aft side guide 284.

FIGS. 13 and 14 show the left gun bolt head assembly 120L, the left side guide assembly 252L, the center guide assembly 250, the left aft stripper 66L and the left forward stripper 64L in their respective feed dispositions. The left gun bolt head 120L is in rear dwell and abuts the aft end of the aft side guide 284 swinging this aft end outwardly (here seen clock-wise) about the pivot 286, and via the spring 288, swinging the forward end of the forward side guide 282 inwardly over the round which is vertically aligned with the left hand gun barrel and still in its link 241 in the transverse guide 236 under the control of the sprocket 242. Prior to this, the gun bolt while moving aft towards rear dwell had over-ridden the cam surface 272L to disengage the detent plate 260 from the left hand slot in the housing, and while the detent plate is disengaged, had abuted the lateral cam surface 276L to swing the center guide assembly 250 about its pivot 253 to its swung-right disposition, whereat the detent plate engages the right hand slot in the housing. The left slider 44L has been moving aftwardly and is now about three-fourths aft. The aft and forward strippers 66L and 64L are below the round.

FIGS. 15 and 16, and also FIG. 10, show the round stripped out of its link by the aft and forward strippers 66L and 64L which were raised by the still aftwardly moving left slider 44L. The left gun bolt head 120L remains in rear dwell and

an upper portion of the base of the cartridge case is now on a lower portion of the face of the gun bolt, with the extractor disk of the case engaged by the lower extractor lugs 144L/R. Upward movement of the base of the case is limited by a pair of mutually facing surfaces 277LL and 277LR, while the shoulder of the case is limited by the lower face of the forward side guide 282L.

FIGS. 17 and 18 show the round being carried forward by the left gun bolt head which is carried forward by the left slider moving forward, via the accelerator link. The round is nosed upwardly at about 1° to the axis of the gun barrel and is guided by the left concave lateral guide surface 280L on the center guide assembly and a concave guide surface 281L in the breech housing 12 which slope upwardly at about 1° to the axis of the gun barrel. An additional pair of mutually facing, upward sloping guide surfaces 281LL and 281LR are positioned further forward on the breech housing 12 to continue guidance of the round as it is moved forward.

FIGS. 19 and 20 show the round chambered and locked into the left gun barrel by the left gun bolt head which has been carried forward by the still moving forward left slider. As the bolt head moved forwardly, the lugs 78L/R of the left forward stripper respectively entered the channels 172L/R grooves in the bolt head, and, thereafter, the left slider pulled the left forward stripper, and therewith the left bolt head, downwardly, into the bolt head locked disposition. Meanwhile, the right gun bolt head moving aftwardly, has disengaged the detent and switched the center guide assembly into its left hand disposition.

FIGS. 21 and 22 show the fired cartridge case during extraction, (after the chambered and locked round has been fired). As the left bolt head moves aftwardly, driven by left slider via the left accelerator link, the extractor disk of the fired cartridge case rides onto the concave upper guide surface 278L of the center guide assembly and is thereby cammed upwardly along the face of the gun bolt so that while initially engaged with the lower extractor lugs 144L/R and aligned with a pair of mutually facing, concave guide surfaces 283LL and 283LR in the breech housing 12 which are parallel to and spaced vertically above the axis of the gun barrel, engagement of the extractor disk transfers to the upper extractor lugs 142L/R as the bolt moves aftwardly. The left bolt head subsequently disengages the detent and shifts the center guide assembly to its right hand disposition as shown in FIG. 23.

FIGS. 23 and 24 show the fired cartridge case further extracted as the left bolt head continues to travel aftwardly. The left bolt head abuts the aft end of the aft side guide 284 swinging this aft end outwardly (here shown clock-wise) about the pivot 286, and via the spring 288, swinging the forward end of the forward side guide 282 inwardly to strike, and spring yield, against the side of the extracted cartridge case, and subsequently to swing in further under and to push upwardly the shoulder of the extracted cartridge

case, and overlies the next round which is to be stripped from its link and is lying on the transverse guide 236. An ejector guide 270 is bolted to the breech housing and limits upward movement of the base of the round on the bolt face.

FIGS. 25 and 26 show the fired cartridge case fully extracted, still engaged with the upper extractor lugs of the left bolt head which is in rear dwell, and its base resting on laterally extending trackways (not visible) in the breech housing 12 and on the base of the next round which has been stripped from its link (much as was shown in FIG. 16) and its shoulder resting on the side guide 282L. The slider is still moving aft and the center guide assembly is in its right hand disposition.

FIGS. 27 and 28 show the fired cartridge case being ejected. The extractor disk of the fired case is pushed upwardly off the face of the gun bolt by the upper face of the forward side guide 282L and an upper pair of mutually facing, concave guide surfaces 307LL and 307LR in the breech housing, as the gun bolt is moving forward from rear dwell in the disposition substantially as shown in FIGS. 17 and 18.

FIGS. 29 and 30 show the hammer assembly 30 in the on-sear or cocked disposition, while FIGS. 31 and 32 show the off-sear or fired disposition. The hammer assembly includes a hammer housing 308 which is secured to the breech housing 12 by two forward bolts 310 and one aft bolt 312. The hammer head 314 is integral with a hammer body 316 which has a longitudinal bore 318 and a substantially downwardly directed double clevis 320 and a forward facing concave cam 322 adapted to receive an aftwardly projecting sear 364. The hammer body 316 is journaled to the housing 308 by a hammer shaft 326 having a main portion which fits the bore 318, a reduced forward portion which fits a forward bore 328 and an enlarged aft portion which fits an aft bore 330.

A firing spring housing 332 is secured to the breech housing, as by mutual threads 334. The housing contains a helical compression firing spring 336, a spring follower 338 having a socket 340, a connecting link 342 having a knuckle 344 on its lower end riding in the socket 340 and a cross-bar 346 on its upper end riding in the double clevis 320.

A solenoid 348 is bolted to the hammer housing 308 and has an armature 350 which is pulled down into the housing when the solenoid is energized so that the gun may fire. The rate of fire may be controlled by pulsed energization of the solenoid. A sear control lever 352 is pivotally mounted to the hammer housing by a cross-pin 354 and has a bifurcated forward end 356 which rides under the head 358 of the armature 350. The aft end of the lever has a nose with a flat face 360 which is substantially perpendicular to its radius to the pivot 354 and also has a sloping face 362. The sear pin 364 is disposed in a bore 366 in the hammer housing, and has a flat step 368 which rides under a cross-pin 370 to prevent the sear pin from rotating but which permits limited fore and aft movement of the sear pin. The sear pin has a

wedge shaped aft end 372 which rides into the forward facing concave cam 322 of the hammer body 316, and has flat transverse face 374 adapted to abut the flat face 360 of the control lever and has a sloping face 376 adapted to be cammed by the sloping face 362 of the control lever. A helical compression spring 378 is captured between the hammer housing 308 and the sear control lever 352 and biases the lever 352 into its on-sear disposition as shown in FIG. 29 when the solenoid 348 is deenergized, so that the sloping surface 362 on the aft end of the lever cams against the sloping surface 376 on the forward end of the sear pin to push the sear pin 364 aftwardly until the lever snaps down so that the faces 360 and 374 are in abutment to hold the sear pin aft. When the sear pin is held aft, its aft wedge shaped end 372 is disposed within the forward facing concave cam 322 of the hammer body 316 and positively interferes with the downwardly directed swing, either clockwise or counter clockwise, of the hammer head 314, which is urged by firing spring 336 through the connecting link 342. However, when the solenoid 348 is energized, and pulls in its armature 350 to swing the forward end 356 of the control lever 352 downwardly against the bias of the sear spring 378 and to swing the aft end flat face 360 out of the path of the sear pin 364, the firing spring biases the hammer head downwardly and the forward facing concave cam 322 cams the wedge shaped aft end 372 of the sear pin 364 forwardly, out of the concavity of the cam 322, so that the hammer head is free to swing downwardly.

The various positions of the hammer head 314 are shown, (looking forward from aft), in FIGS. 33 through 37. FIG. 33 shows the hammer head, at the time of firing the right hand gun barrel, impacting the head 160R of the right hand firing pin 158R of the right hand gun bolt head assembly 120R. The hammer head is clear of the right forward stripper 64R, the left forward stripper 64L, and the head 160L of the left hand firing pin 158L.

The hammer head has a left hand safety sear surface 382L and a right hand safety sear surface 382R whose respective function is to ensure that the respective gun bolt head is safely locked, i.e., the locking lugs 173a, b, c and on the aft face of the gun bolt head and the respective locking lugs 174a, b, c and d on the breech housing are adequately, but not totally, mutually engaged, before the hammer head is otherwise clear to be driven fully downwardly to impact the respective firing pin.

FIG. 34 shows the right hand gun bolt fired and fully unlocked. The hammer upper camming surface 79U of the right forward stripper 64R engages a right hand camming surface 380R and as the stripper is raised by the slider 44R, the stripper cams the hammer counter-clockwise against the bias of the firing spring 336. As the hammer upper camming surface loses engagement with the hammer, the stripper lower camming surface 79D engages the right hand safety

sear surface 382R and continues to cam the hammer counter-clockwise as shown in FIG. 35 until the hammer goes over-center as shown in FIG. 36, and the firing spring 336 drives the hammer further counter-clockwise until the safety sear surface 382L arrests further rotation until the gun bolt is safely locked as described above, after which the hammer is driven further counter-clockwise to impact the head 160L of the left hand firing pin 158L of the left hand gun bolt head assembly 120L as shown in FIG. 37.

The gas drive and clearing assembly are shown in their deenergized or cleared disposition in FIGS. 38 and 39, and also shown in part in FIG. 4. A detent 386, which is mounted in the receiver housing 26 to slide fore and aft, has an aftwardly extending projection 388 with a depending foot 390 which projects into a recess 392 in the rocking lever 34. The recess 392 has an inner arcuate wall 394 which has a left and a right locking surface 396L/R. The foot 390 has a forward facing locking surface 398 which rides against and along the surfaces 394, 396L and 396R. The rocking lever 34 also has a peripheral ridge 400 which terminates in two radially projecting blocking faces 402L/R which are respectively adapted to abut against either of two blocking surfaces 404L/R on the detent when the locking surface 398 is wedged against either locking surface 396L or 396R. Thus, when the rocking lever 34 is swung fully counter-clockwise beyond the disposition shown in FIG. 4, the detent surface 398 is cammed onto the surface 396R and the detent is pulled aftward and is simultaneously pushed aftward by three parallel springs 446 so that the peripheral face 402R is wedged against the detent surface 404L. For the face 402R to ride off the surface 404L, the detent must move forward. If such forward movement is precluded, the rocking lever is frictionally locked against rotation, and operation of the operating mechanism of the gun is halted. When the rocking lever is swung fully clockwise, and the detent surface 398 is cammed onto the surface 396L, a similar locking action is provided.

The gas drive and clearing assembly includes a left and a right cylinder 408L/R, each cylinder having a respective piston rod 410L/R, each rod having a respective aft end formed at the disk 92L/R, each disk adapted to fit the respective slot 90L/R in the forward end of the respective slider 44L/R. The forward ends of the cylinder are connected by a manifold block 414 which has a socket to receive a gas squib, or a plurality of squibs (not shown), whose upper end 416 is threaded to receive an electrical connector to enable the firing of the squib to provide gas under pressure in the manifold and thence to the cylinders. A rocker arm 418 is pivotally mounted at 420 to the manifold and is coupled to two pilot rods 422L/R which are respectively activated by the piston rods 94L/R at the forward end of their stroke to open the respective valve (not shown) in the manifold. Thus, when the right hand piston rod 94R is forward, the right hand pilot rod 422R is driven forward and opens the right hand valve to

permit the flow of gas from the squib into the right hand piston 408R and swings the rocker arm 418 to close the left hand valve to preclude the flow of gas into the left hand piston 408L. The pistons also have respective gas bleed ports 424L/R which are aligned with corresponding gas bleed ports, not shown, in the gun barrels. The ports 424 are not controlled by the valves in the manifold and pass gas directly from the fired gun barrel into the respective gas cylinder to drive the respective piston aftwardly to drive the respective slider aftwardly. The cylinders have respective forward mounting brackets, 426L/R which are bolted to a top clamp bracket 428 by four bolts to form the forward clamp assembly. The cylinders have respective aft mounting brackets 425aL/R which are bolted to a bracket 430 which is bolted to a top clamp bracket 432.

A clearing lever 434 has a clevis 436 at its forward end pivotally mounted at 438 to a tab depending from the bracket 430 and has a depending foot 440 at its aft end projecting through a slot 442 in the forward end of the receiver housing 26 adjacent the forward end of the detent 386. The detent 386 has three blind bores 444 therein, in each of which is disposed a respective helical compression spring 446 which is captured between the detent and the receiver housing and which biases the detent aftwardly. A clearing solenoid 448 is bolted to the bracket 430 and has an armature 450, with a notch 452 in its distal end which captures the clearing lever 434, and a helical compression spring 454 which biases the clearing lever and the armature downwardly. When the clearing lever 434 is biased downwardly, as shown in FIGS. 38 and 39, the depending foot 440 is disposed immediately forward of the detent 386 and precludes the detent from moving forwardly, which precludes the rocking lever from rotating, and thereby halts the operation of the gun, notwithstanding any gas pressure in either of the cylinders 408L/R, thus "safing" the gun by leaving a fired cartridge case in one of the barrels. When the clearing solenoid 448 is energized, as shown in FIGS. 40 and 41, it pulls up the armature 450 and the clearing lever 434, against the bias of the spring 454, to swing the depending foot 440 upwardly and out of the way of the detent 386, to permit the detent 386 to move forwardly against the bias of the springs 446, when gas pressure in either of the cylinders tends to cause the rocking lever to rotate, so that the detent thereby permits the rocking lever to rotate, thereby permitting operation of the gun. In this disposition, gas pressure in the pistons will reciprocate the sliders 44L/R, which will (1) oscillate the rocking lever 34, which will reciprocate the feeder arm 48, which will reciprocate the cam driver 58 which will drive the feeder sprocket, and (2) reciprocate the strippers and the gun bolt assemblies. If rounds have (1) been fed and stripped and loaded and locked, and (2) if the firing solenoid 348 has been energized, then the gun will fire.

If it is desired to have the gun powered by an

external source, rather than self-powered, as by the gas drive shown in FIGS. 38 and 40, then an external drive may be substituted as shown in FIGS. 42 and 43. The safing system, which includes the clearing solenoid 448, the clearing lever 434 and the detent 386, remains as shown in FIG. 39 to provide their functions. The external drive assembly includes an electric motor 460, fixed to the receiver housing, whose output shaft 462 is fixed to a shaft 464 which is fixed to a lower bevel gear 466. The gear 466 is meshed with a lateral bevel gear 468 fixed to a shaft 469 and which is meshed with an upper bevel gear 470, which is journaled for rotation with respect to the shaft 464. A lower cam roller 472 is journaled for rotation on a stub shaft 474 which is carried by the lower gear 466. An upper cam roller 476 is journaled for rotation on a stub shaft 478 which is carried by the upper gear 470. A shaft 480 is splined to the central post 36, shown in FIG. 4, which post is fixed to the rocking lever 34. A bifurcated cam follower 482 is fixed to the shaft 480 and its bifurcation projects between the gears 466 and 470 so that the rollers 472 and 476 may ride into and out of the cleft cam following surface 484 of the bifurcation. The shafts 464, 469, and 480 are each respectively supported by stationary journals, (not shown).

As viewed in FIG. 42, the electric motor 460 drives the shaft 464 with the gear 466 counter-clockwise so that the roller 472 is carried counter-clockwise and swings the bifurcated cam follower 482 with the shaft 480 and the rocking lever 34 clockwise until the roller 472 exits the cleft 484 and the swing of the follower 482 halts. Meanwhile, the gears 466 and 468 drive the gear 470 clockwise until it enters the cleft 484 and swings the follower 482, and thereby, the rocking lever 34 counter-clockwise until the roller 476 exits the cleft 484 and the swing of the follower 482 halts until the roller 482 reenters the cleft, and so on.

Alternatively, an external drive may be substituted as shown in FIG. 44. This external drive assembly includes a double acting cylinder 500 having a piston 502 whose forward end is connecting by a pivotable joint 504 to a journal rod 506 which is journaled on a post 508 which is fixed to and depends from the underside of the rocking lever 34 and is eccentric to the central post 36. Aftward movement of the piston 502 pulls the rocking lever through the disposition shown in FIG. 44 to bring the left slider 44L into its aft disposition. Forward movement of the piston 502 pushes the rocking lever to bring the right slider 44R into its aft disposition.

Claims

1. A machine gun comprising:
 - a receiver (12; 26)
 - a pair of gun barrels (10L; 10R) fixed to said receiver (12; 26)
 - a pair of operating mechanisms, and a pair of gun bolt assemblies (118LR) including a pair of

gun bolts, disposed in said receiver (12, 26) for loading, locking and firing each of said gun barrels (10/R) respectively;

a rocking lever (34) coupled to and between said operating mechanisms for driving said mechanisms 180° out of mutual phase;

said rocking lever (34) journaled for pivotal movement about a central axis (36); and

external driving means comprising a motor (460) coupled to said rocking lever (34) for reciprocating said lever about said axis (36) characterized in that

said external drive means includes:

a pinion gear (468) driven by the motor (460);

said pinion gear (468) meshed with and driving two mutually opposed gears (466, 470), respectively journaled for rotation about a common axis (460), in opposite directions of rotation;

a cam follower (482) having a cleft cam surface (484) disposed between said two opposed gears (466, 470) and journaled for oscillation about an axis (480) to and between two angularly spaced apart dispositions;

two cams (472, 476) each carried by a respective one of said two opposed gears (466, 470); and one of said cams (472) adapted to enter said cleft (484) at one of said dispositions and to swing said follower (482) to the other of said dispositions and to exit said cleft, and the other of said cams (476) adapted to enter said cleft at said other of said dispositions and to swing said follower to said one of said dispositions and to exit said cleft.

2. A machine gun as in claim 1 characterized in that each of said operating mechanisms includes:

a slider (44L/R) coupled to said rocking lever (34) for fore and aft rectilinear reciprocation as said lever (34) reciprocates about its said central axis (36); and

a forward stripper (64L/R) and an aft stripper (66L/R) each adapted to be cammed transversely up and down by said slider (44L/R) during the course of said reciprocation or said slider (44L/R); said aft and forward strippers (64L/R; 66L/R) adapted to strip rounds of ammunition from a linked belt of ammunition;

said forward stripper (64L/R) being adapted to lock said respective gun bolt assembly (118L/R).

3. A machine gun according to claim 2 characterized in that

said pair of gun bolts are respectively coupled to a pair of accelerating linkages (124L/R), which are each respectively coupled to said respective slider (44L/R) in said pair of operating mechanisms, whereby said respective slider (44L/R) drives said coupled thereto gun bolt fore and aft to load a round of ammunition, and said respective forward stripper (64L/R) locks and unlocks said gun bolt, during the course of said reciprocation of said slider (44L/R).

4. A machine gun according to claim 3 characterized in that

each gun bolt has a bolt head (120L/R) which carries a respective firing pin (158) which is mounted on a pivot (166) within a cavity (128) in

said bolt head (120L/R) on a pivotal axis which is transverse to the longitudinal axis of the respective gun barrel (10L/R);

said firing pin (158) has a head (160) projecting aftwardly from said pivot and a leg (168) depending from said pivot (166) and terminating in a penetrator tip (170) which projects forwardly through a bore (138) leading from said cavity (128) to the face of the gun bolt.

5. A machine gun according to claim 4 further characterized in that it comprises:

a hammer assembly (30) having a hammer head (314) adapted to swing to and between one disposition whereat it strikes the head of one of said firing pins (138) when the respective one bolt head (120L/R) is in its locked disposition and another disposition whereat it strikes the head of the other of said firing pins (158) when the respective other bolt head (120L/R) is in its locked disposition;

each firing pin (158) when struck by said hammer being adapted to swing its penetrator (138) forwardly beyond the face (140) of the gun bolt into the primer cup of a round of ammunition captured on the face (140) of the gun bolt;

said penetrator (138) having a taper such that it forms a tapered cavity in the primer cup whose surface will cam the penetrator (138) aftwardly behind said bolt face (140) upon transverse movement of the round along said bolt face (140).

6. A machine gun as in claim 2 characterized in that

each gun bolt assembly (118L/R) is coupled to said slider (44L/R) and mounted for fore and aft reciprocation in response to said reciprocation of said slider; and

the forward strippers (64L/R) act as locking elements cooperating with said slider (44L/R) and mounted for transverse reciprocation in response to said reciprocation of said slider (44L/R) for engaging and disengaging said gun bolt assembly (118L/R) in a forward disposition thereof and for transversely shifting said gun bolt assembly (118L/R) to and between an unlocked and locked disposition.

7. A machine gun according to claim 6 characterized in that it further includes

a sprocket (242) for advancing a belt of linked ammunition; and in that in each operating mechanism

said forward stripper (64L/R) and said aft stripper (66L/R), both cooperate with said slider (44L/R) and are mounted for transverse reciprocation in response to said reciprocation of said slider (44L/R) for stripping a round of ammunition from a belt of linked ammunition in said sprocket (242).

8. A machine gun according to claim 7 characterized in that

the pitch of the tips of the teeth of said sprocket (242) is greater than the pitch of the belt of linked ammunition.

9. A machine gun according to claim 6 characterized in that

said forward strippers (64L/R) block said hammer heads (314) from striking its respective bolt head (120L/R) until said forward strippers (64L/R) have engaged and shifted its respective bolt head (120L/R) to its locked disposition.

10. A machine gun according to claim 7 characterized in that

each of said gun bolts is coupled to a respective accelerating linkage (124L/R) by a quick-release pivotal connection (178) including

a clevis (206) and a crosspin (212) formed in an end of one of the components comprising the group of bolt and accelerating linkage (124L/R), and

a hook (216) and a spring loaded closure (224, 226) therefor, both formed in an end of the other of the components in said group, said hook (216) adapted to enter said clevis (206) and engaged onto said crosspin (212), said closure (224; 226) adapted to capture said hook onto said crosspin.

11. A machine gun according to claim 9 characterized in that

said sprocket (242) is mounted to a first shaft (244);

said first shaft (244) is releasably coupled to and driven by a second shaft (245);

said second shaft (245) is coupled by a one-way clutch (247) to a drum cam (246);

said drum cam (246) is driven by a cam driver assembly (58) which is coupled to and driven by said rocking lever (34),

whereby oscillation of said rocking lever (34) intermittently rotates said sprocket (242) in a first direction to advance the belt of ammunition, and decoupling said first shaft (244) from said second shaft (245) permits rotation of said sprocket (242) in a second direction opposite to said first direction.

12. A machine gun as in any of the preceding claims characterized in that it further comprises a clearing mechanism including:

detent means (386) having a first disposition whereat said detent means (386) engages said rocking lever (34) to preclude rocking thereof and a second disposition permitting rocking thereof;

spring means (446) normally biasing said detent means (386) to its first disposition;

blocking means (434) having a first disposition whereat said blocking means (434) blocks said detent means (386) from shifting from said first disposition to said second disposition;

additional spring means (454) normally biasing said blocking means (434) to its first disposition;

solenoid means (448) having a deenergized disposition for permitting said blocking means (434) to be disposed by said additional spring means (454) in its first disposition and having an energized disposition for shifting said blocking means (434) to its second disposition.

13. A gun according to claim 1 characterized in that

said cam follower (482) is fixed to a shaft (480) which is fixed to a first gear which is meshed with a second gear which is journaled for oscillation about said central axis of said rocking lever

and connected to said rocking lever (34); having a mode of operation that

continuous rotation of said motor provides rotation of said two cams (472, 476) in respective opposite directions, one of said cams swinging said follower in one direction and the other of said cams swing said follower in the other direction, and such oscillation of said follower providing oscillation of said rocking lever (34).

14. A gun according to claim 13 characterized in that said second gear is connected to said rocking lever (34) by a common shaft fixed to both.

15. A gun according to claim 13 characterized in that said second gear is integral with said rocking lever.

Patentansprüche

1. Maschinengeschütz enthaltend:
einen Aufnehmer (12; 26),
ein Paar Geschützläufe (10L; 10R), die an dem Aufnehmer (12; 26) befestigt sind,
ein Paar Betätigungsmechanismen und ein Paar Geschützverschlußanordnungen (118L/R) mit einem Paar Geschützverschlüssen, die in dem Aufnehmer (12; 26) angeordnet sind, zum Laden, Verriegeln und Feuern von jedem der Geschützläufe (10L/R) auf entsprechende Weise,
einen Schwenkhebel (34), der mit und zwischen den Betätigungsmechanismen verbunden ist zum Antreiben der Mechanismen 180° aus der gegenseitigen Phase,
wobei der Schwenkhebel (34) für eine Schwenkbewegung um eine Mittelachse (36) gelagert ist, und
eine externe Antriebsvorrichtung mit einem Motor (460), der mit dem Schwenkhebel (34) verbunden ist für eine Hin- und Herbewegung des Hebels um die Achse (36),
dadurch gekennzeichnet, daß
die externe Antriebsvorrichtung aufweist:
ein Antriebsritzel (468), das durch den Motor (460) angetrieben ist,
wobei das Antriebsritzel (468) mit zwei zueinander gegenüberliegenden Zahnradern (466, 470) kämmt und antreibt, die auf entsprechende Weise für eine Rotation um eine gemeinsame Achse (460) in entgegengesetzten Drehrichtungen gelagert sind,
einen Nockenfolger (482) mit einer geschlitzten Nockenfläche (484), der zwischen den zwei gegenüberliegenden Zahnradern (466, 470) angeordnet und für eine Schwenkbewegung um eine Achse (480) in und zwischen zwei mit Winkelabstand angeordneten Lagen angeordnet ist,
zwei Nocken (472, 476), die jeweils von einem entsprechenden der zwei gegenüberliegenden Zahnradern (466, 470) getragen sind, wobei die eine Nocke (472) in einer der Lagen in den Schlitz (482) eintreten und den Folger (482) in die andere Lage schwenken und aus dem Schlitz austreten kann, und die andere Nocke (476) in der anderen Lage in den Schlitz eintreten und den Folger in die eine Lage schwenken und aus dem Schlitz austreten kann.

2. Maschinengeschütz nach Anspruch 1, dadurch gekennzeichnet, daß jeder Betätigungsmechanismus aufweist:

einen Gleiter (44L/R), der mit dem Schwenkhebel (34) verbunden ist für eine geradlinige Hin- und Herbewegung nach vorn und hinten, wenn der Hebel (34) sich um seine Mittelachse (36) hin- und herbewegt, und

einen vorderen Abstreifer (64L/R) und einen hinteren Abstreifer (66L/R), die während der Hin- und Herbewegung des Gleiters (44L/R) jeweils durch den Gleiter (44L/R) quer nach oben und unten betwegt werden können,

wobei die hinteren und vorderen Abstreifer (64L/R; 66L/R) Patronen von einem verdunnenen Munitionsgurt abstreifen können,

der vordere Abstreifer (64L/R) die entsprechende Geschützverschlußanordnung (118L/R) verriegeln kann.

3. Maschinengeschütz nach Anspruch 2, dadurch gekennzeichnet, daß

das Parr Geschützverschlüsse auf entsprechende Weise mit einem Parr Beschleunigungsgestängen (124L/R) verdunnen ist, die jeweils auf entsprechende Weise mit dem entsprechenden Gleiter (44L/R) in den zwei Betätigungsmechanismen verbunden sind, wobei der entsprechende Gleiter (44L/R) den damit verbundenen Geschützverschluß nach vorne und hinten antreibt, um eine Patrone zu laden, und der entsprechende vordere Anstreicher (64L/R) den Geschützverschluß im Laufe der Hin- und Herbewegung des Gleiters (44L/R) verriegelt und entriegelt.

4. Maschinengeschütz nach Anspruch 3, dadurch gekennzeichnet, daß

jeder Geschützverschluß einen Verschlußkopf (120L/R) aufweist, der einen entsprechenden Zündstift (158) trägt, der auf einem Drehzapfen (166) in einer Kammer (128) in dem Verschlußkopf (120L/R) auf einer Schwenkachse angebracht ist, die quer zu der Längsachse des entsprechenden Geschützlaufes (10L/R) verläuft,

wobei der Zündstift (158) einen Kopf (160), der von dem Drehzapfen nach hinten ragt, und einen Schenkel (168) aufweist, der von dem Drehzapfen (166) herabhängt und in einer Durchringungsspitze (170) endet, die durch eine Bohrung (138) nach vorne ragt, welche von der Kammer (128) zu der Stirnfläche des Geschützverschlusses führt.

5. Maschinengeschütz nach Anspruch 4, gekennzeichnet durch:

eine Hammeranordnung (30) mit einem Hammerkopf (314), der in und zwischen einer Lage, in der er auf den Kopf von einem der Zündstifte (138) trifft, wenn der entsprechende Verschlußkopf (120L/R) in seiner verriegelten Lage ist, und einer anderen Lage schwenkbar ist, in der er auf den Kopf von dem anderen Zündstift (158) trifft, wenn der entsprechende andere Verschlußkopf (120L/R) in seiner verriegelten Lage ist,

wobei jeder Zündstift (158), wenn er von dem Hammer getroffen wird, seinen Durchdringer (138) nach vorne über die Stirnfläche (140) des

Geschützverschlusses hinaus in das Zündhütchen einer Patrone schwenken kann, die auf der Fläche (140) des Geschützverschlusses eingeschlossen ist,

wobei der Durchdringer (138) einen Konus aufweist derart, daß er eine konische Kammer in dem Zündhütchen bildet, deren Oberfläche den Durchdringer (138) nach hinten leitet hinter die Verschußfläche (140) bei einer Querbewegung der Patrone entlang der Verschußfläche (140).

6. Maschinengeschütz nach Anspruch 2,

dadurch gekennzeichnet, daß

jede Geschützverschußanordnung (118L/R) mit dem Gleiter (44L/R) vergunden und für eine Hin- und Herbewegung nach vorne und hinten angebracht ist bei einer Hin- und Herbewegung des Gleiters, und

die vorderen Abstreifer (64L/R) als Verriegelungselemente wirken, die mit dem Gleiter (44L/R) zusammenarbeiten und für eine Hin- und Herbewegung in Querrichtung bei der Hin- und Herbewegung des Gleiters (44L/R) angebracht sind zum Ergriffen und Lösen der Geschützverschußanordnung (118L/R) in ihrer vorderen Lage und zum Querverschieben der Geschützverschußanordnung (118L/R) in und zwischen einer entriegelten und verriegelten Lage.

7. Maschinengeschütz nach Anspruch 6,

dadurch gekennzeichnet, daß

eine Transportzahnrad (242) vorgesehen ist zum Vorwärtsbewegen eines Gurtes verbundener Patronen und daß in jedem Betätigungsmechanismus

der vordere Abstreifer (64L/R) und der hintere Abstreifer (44L/R) sowohl mit dem Gleiter (44L/R) zusammenarbeiten als auch für eine Hin- und Herbewegung in Querrichtung angebracht sind bei der Hin- und Herbewegung des Gleiters (44L/R), um eine Patrone aus dem Patronengurt in das Transportzahnrad (242) abzustreifen.

8. Maschinengeschütz nach Anspruch 7,

dadurch gekennzeichnet, daß

die Steigung der Spitzen der Zähne des Transportzahnrades (242) größer ist als die Steigung des Patronengurtes.

9. Maschinengeschütz nach Anspruch 6,

dadurch gekennzeichnet, daß

die vorderen Abstreifer (64L/R) die Hammerköpfe (314) an einem Aufprall auf ihren entsprechenden Verschußkopf (120L/R) hindern, bis die vorderen Abstreifer (64L/R) ihren entsprechenden Verschußkopf (120L/R) ergriffen und in seine verriegelte Lage geschoben haben.

10. Maschinengeschütz nach Anspruch 7,

dadurch gekennzeichnet, daß

jeder der Geschützverschlüsse mit einem entsprechenden beschleunigenden Gestänge (124L/R) durch eine schnell zu lösende Schwenkverbindung (178) verbunden ist, die

einen Gabelkopf (206) und einen Querstift (212), die in einem Ende von einer der Komponenten ausgebildet sind, die die Verschußgruppe und das beschleunigende Gestänge (124L/R) enthalten, und

einen Haken (216) und einen Feder-belasteten

Verschuß (224, 226) dafür aufweist, die beide in einem Ende der anderen Komponenten in der Gruppe ausgebildet sind, wobei der Haken (216) in den Gabelkopf (206) eintreten und den Querstift (212) ergreifen kann und der Verschuß (224, 226) den Haken auf dem Querstift einschließen kann.

11. Maschinengeschütz nach Anspruch 9,

dadurch gekennzeichnet, daß

das Transportzahnrad (242) an einer ersten Welle (244) angebracht ist,

die erste Welle (244) mit einer zweiten Welle (245) lösbar verbunden und durch diese antreibbar ist,

die zweite Welle (245) durch eine Einwegkuppelung (247) mit einer Trommelnocke (246) verbunden ist,

die Trommelnocke (246) durch eine Nocken-antriebsvorrichtung (58) angetrieben ist, die mit dem Schwenkhebel (34) verbunden und von diesem angetrieben ist,

wobei die Schwenkbewegung des Schwenkhebels (34) das Transportzahnrad (242) intermittierend in eine erste Richtung dreht, um den Munitionsgurt vorwärtszuschieben, und die Trennung der ersten Welle (244) von der zweiten Welle (245) eine Drehung des Transportzahnades (242) in einer zweiten Richtung engengesetzt zur ersten Richtung gestattet.

12. Maschinengeschütz nach einem der vorstehenden Ansprüche,

dadurch gekennzeichnet, daß

ein Freigabemechanismus vorgesehen ist, der enthält:

eine Klinkenvorrichtung (386) mit einer ersten Lage, in der die Klinkenvorrichtung (386) den Schwenkhebel (34) ergreift, um seine Schwenkbewegung zu verhindern, und mit einer zweiten Lage, in der die Schwenkbewegung gestattet ist,

Federmittel (446), die die Klinkenvorrichtung (386) normalerweise in ihre erste Lage vorspannen,

eine Blockiervorrichtung (434) mit einer ersten Lage, in der die Blockiervorrichtung (434) eine Verschiebung der Klinkenvorrichtung (386) von der ersten Lage in die zweite Lage blockiert,

zusätzliche Federmittel (454), die die Blockiervorrichtung (434) normalerweise in ihre erste Lage vorspannen,

eine Magnetspulenvorrichtung (448) mit einer ausgeschalteten Lage, damit die Blockiervorrichtung (434) durch die zusätzlichen Federmittel (454) in ihre erste Lage gebracht werden kann, und mit einem eingeschalteten Zustand zum Verschieben der Blockiervorrichtung (43) in ihre zweite Lage.

13. Geschütz nach Anspruch 1,

dadurch gekennzeichnet, daß

der Nockenfolger (482) an einer Welle (480) befestigt ist, die an einem ersten Zahnrad befestigt ist, das mit einem zweiten Zahnrad kämmt, das für eine Schwenkbewegung um die Mittelachse des Schwenkhebels gelagert und mit dem Schwenkhebel (34) verbunden ist, und daß in einem Betriebsmodus eine kontinuierliche Rotation des Motors für eine Drehung der zwei Nocken

(472, 476) in entsprechende entgegengesetzte Richtungen sorgt, wobei die eine Nocke den Folger in die eine Richtung und die andere Nocke den Folger in die andere Richtung schwenken und diese Schwenkbewegung des Folgers für eine Schwenkbewegung des Schwenkhebels (34) sorgt.

14. Geschütz nach Anspruch 13, dadurch gekennzeichnet, daß das zweite Zahnrad mit dem Schwenkhebel (34) durch eine gemeinsame Welle verbunden ist, die an beiden befestigt ist.

15. Geschütz nach Anspruch 13, dadurch gekennzeichnet, daß das zweite Zahnrad einstückig mit dem Schwenkhebel ausgebildet ist.

Revendications

1. Mitrailleur comprenant:
un récepteur (12; 26)
une paire de tubes (10L; 10R) fixés au récepteur (12; 26)

une paire de mécanismes de manoeuvre, et une paire d'ensembles à culasse (118L/R) comportant une paire de culasses, disposés dans le récepteur (12, 26) pour la charge, le verrouillage et le tir de chacun des tubes (10/R) respectivement;

un levier basculant (34) accouplé aux mécanismes de manoeuvre, entre ceux-ci, pour entraîner lesdits mécanismes à un angle de déphasage de 180° entre eux;

le levier basculant (34) étant monté pour un mouvement pivotant autour d'un axe central (36); et

un moyen d'entraînement extérieur comportant un moteur (460) accouplé au levier basculant (34) pour animer d'un mouvement de va-et-vient le levier autour de l'axe (36), caractérisé en ce que:

le moyen d'entraînement extérieur comporte:

un pignon (468) entraîné par le moteur (460);

le pignon (468) étant en prise avec deux engrenages mutuellement opposés (466, 470) pour les entraîner, engrenages montés respectivement pour rotation autour d'un axe commun (460), dans des directions de rotation opposées;

un galet de came (482) ayant une surface à came à interstice (484) disposée entre les deux engrenages opposés (466, 470) et montés pour osciller autour d'un axe (480) jusqu'à et entre deux positions espacées angulairement l'une de l'autre;

deux cames (472, 276) supportées chacune par un engrenage respectif des deux engrenages opposés (466, 470); et l'une des cames (472) étant destinée à entrer dans l'interstice (484) à l'une des dites positions et à basculer le galet (482) jusqu'à l'autre position et à le faire sortir de l'interstice, et l'autre came (476) étant destinée à entrer dans l'interstice à l'autre position et à basculer le galet jusqu'à la première des positions et à sortir de l'interstice.

2. Mitrailleur selon la revendication 1, caractérisée en ce que:

chacun des mécanismes de manoeuvre comporte:

une glissière (44L/R) accouplée au levier basculant (34) pour un mouvement alternatif rectiligne vers l'avant et vers l'arrière alors que le levier (34) est animé d'un mouvement de va-et-vient autour de son axe (36); et

un éjecteur avant (64L/R) et un éjecteur arrière (66L/R) destinés chacun à être soumis à un effet de came dans le sens transversal vers le haut et vers le bas par la glissière (44L/R) pendant la course de va-et-vient de la glissière (44L/R);

les éjecteurs arrière et avant (64L/R; 66L/R) étant destinés à détacher des cartouches d'une bande articulée de munitions;

l'éjecteur avant (64L/R) étant destiné à verrouiller l'ensemble respectif à culasse (118L/R).

3. Mitrailleur selon la revendication 2, caractérisée en ce que:

la paire de culasses est respectivement accouplée à une paire de biellettes d'accélération (124L/R), qui sont chacune accouplée respectivement à la glissière respective (44L/R) de la paire de mécanismes de manoeuvre, d'où il résulte que la glissière respective (44L/R) entraîne vers l'avant et vers l'arrière la culasse ainsi accouplée afin de charger une cartouche, et l'éjecteur avant respectif (64L/R) verrouille et déverrouille la culasse, pendant la course de va-et-vient de la glissière (44L/R).

4. Mitrailleur selon la revendication 3, caractérisée en ce que:

chaque culasse comporte une tête de culasse (120L/R) qui supporte un percuteur respectif (158) qui est monté sur un pivot (166) à l'intérieur d'une cavité (128) ménagée dans la tête de culasse (120L/R) sur un axe de pivotement qui est transversal à l'axe longitudinal du tube respectif (10L/R);

le percuteur (158) comporte une tête (160) en saillie vers l'arrière par rapport au pivot et une branche solidaire du pivot (166) et se terminant par une extrémité de pénétrateur (170) qui est en saillie vers l'avant dans un alésage (138) allant de la cavité (128) jusqu'à la face de la culasse.

5. Mitrailleur selon la revendication 4, caractérisée en ce qu'elle comprend en outre:

un ensemble à marteau percutant (30) comportant une tête de marteau (314) destinée à basculer jusqu'à et entre une position où elle heurte la tête de l'un des percuteurs (138) lorsque la tête respective de culasse (120L/R) se trouve dans sa position verrouillée et une autre position dans laquelle elle frappe la tête de l'autre percuteur (158) lorsque l'autre tête respective de culasse (120L/R) se trouve dans sa position verrouillée;

chaque percuteur (158) lorsqu'il est heurté par le marteau percutant étant amené à faire basculer son pénétrateur (138) vers l'avant au delà de la face (140) de la culasse pour qu'il entre dans la coupelle de l'amorce d'une cartouche emprisonnée sur la face (140) de la culasse;

le pénétrateur (138) ayant un chanfrein tel qu'il forme une cavité conique dans la coupelle de l'amorce dont la surface aura un effet de came sur

le pénétrateur (138) dans la direction de l'arrière, à l'arrière de la face (140) de la culasse, lors du mouvement transversal de la cartouche le long de la face (140) de la culasse.

6. Mitrailleuse selon la revendication 2, caractérisée en ce que:

chaque ensemble à culasse (118L/R) est accouplé à la glissière (44L/R) et monté pour être animé d'un mouvement de va-et-vient vers l'avant et vers l'arrière en réponse au mouvement alternatif de la glissière; et

les éjecteurs avant (64L/R) agissent en éléments de verrouillage qui coopèrent avec la glissière (44L/R) et montés pour mouvement alternatif transversal en réponse au mouvement de va-et-vient de la glissière (44L/R) pour engager et désengager l'ensemble à culasse (118L/R) dans sa position avant et pour décaler transversalement l'ensemble à culasse (118L/R) jusqu'à et entre une position déverrouillée et une position verrouillée.

7. Mitrailleuse selon la revendication 6, caractérisée en ce qu'elle comporte en outre:

un pignon (242) pour faire avancer une courroie de munitions articulée; et en ce que dans chaque mécanisme de manœuvre:

l'éjecteur avant (64L/R) et l'éjecteur arrière (66L/R) coopèrent tous deux avec la glissière (44L/R) et sont montés pour mouvement alternatif transversal en réponse au mouvement de va-et-vient de la glissière (44L/R) pour détacher une cartouche d'une bande de munitions articulée dans le pignon (242).

8. Mitrailleuse selon la revendication 7, caractérisée en ce que:

le pas des extrémités des dents du pignon (242) est supérieur au pas de la bande articulée de munitions.

9. Mitrailleuse selon la revendication 2, caractérisée en ce que:

les éjecteurs avant (64L/R) empêchent que les têtes (314) des marteaux percutants ne heurtent leur tête respective de culasse (120L/R) jusqu'à ce que les éjecteurs avant (64L/R) soient venus en contact et aient décalé leur tête de culasse respective (120L/R) jusqu'à la position verrouillée.

10. Mitrailleuse selon la revendication 7, caractérisée en ce que:

chacune des culasses est accouplée à une bielle respective d'accélération (124L/R) par une liaison pivotante à libération rapide (178) comportant:

un étrier (206) et un axe transversal (212) formés dans une extrémité de l'un des composants comprenant le groupe formé par une culasse et une bielle d'accélération (124L/R) et un crochet (216) et une fermeture (224, 226) soumise à l'action d'un ressort destinée au crochet, tous deux étant formés dans une extrémité de l'autre des composants du dit groupe, le crochet (216) étant destiné à entrer dans l'étrier (206) et à venir en contact avec l'axe transversal (212), la fermeture (224, 226) étant destinée à emprisonner le crochet sur l'axe transversal.

11. Mitrailleuse selon la revendication 9, caractérisée en ce que:

le pignon (242) est monté sur un premier arbre (244);

le premier arbre (244) est accouplé de manière amovible avec un second arbre (245) qui l'entraîne;

le second arbre (245) est accouplé par un embrayage unidirectionnel (247) à une came à tambour (246);

la came à tambour (246) est entraînée par un ensemble (58) d'entraînement à came, qui est accouplé au levier basculant (34) et entraîné par celui-ci

d'où il résulte que l'oscillation du levier basculant (34) fait tourner par intermittence le pignon (242) dans une première direction pour faire avancer la bande de munitions, et le désaccouplement du premier arbre (244) du second arbre (245) permet la rotation du pignon (242) dans une seconde direction opposée à la première.

12. Mitrailleuse selon l'une quelconque des revendications précédentes, caractérisée en ce qu'elle comporte en outre:

un mécanisme de dégagement comprenant:

un moyen d'arrêt (386) ayant une première position dans laquelle le moyen d'arrêt (386) est en contact avec le levier basculant (34) pour empêcher son mouvement de basculement et une seconde position permettant son basculement;

un moyen de ressort (446) sollicitant normalement le moyen d'arrêt (386) jusqu'à sa première position;

un moyen de verrouillage (434) ayant une première position dans laquelle le moyen de verrouillage (434) empêche que le moyen d'arrêt (386) ne passe de la première position à la seconde position;

un moyen supplémentaire de ressort (454) sollicitant normalement le moyen de verrouillage (434) vers sa première position;

un moyen de solénoïde (448) ayant une position désexcitée pour permettre au moyen de verrouillage (434) d'être mis par le moyen supplémentaire de ressort (454) dans sa première position et ayant une position excitée pour faire passer le moyen de verrouillage (43) à sa seconde position.

13. Mitrailleuse selon la revendication 1, caractérisée en ce que:

le galet de came (482) est fixé à un arbre (480) qui est assujéti à un premier engrenage en prise avec un second engrenage qui est monté pour oscillation autour du dit axe du levier basculant et relié au levier basculant (34), ayant un mode fonctionnement tel que:

la poursuite de la rotation du moteur provoque la rotation de deux cames (472, 476) dans des directions respectivement opposées, l'une des cames faisant basculer le galet dans une direction et l'autre faisant basculer le galet dans l'autre direction, et une telle oscillation du galet provoque l'oscillation du levier basculant (34).

14. Mitrailleuse selon la revendication 13, caractérisée en ce que le second engrenage est relié au levier basculant (34) par un arbre com-

mun fixé aux deux.

15. Mitrailleuse selon la revendication 13,

caractérisée en ce que le second engrenage est en une pièce avec le levier basculant.

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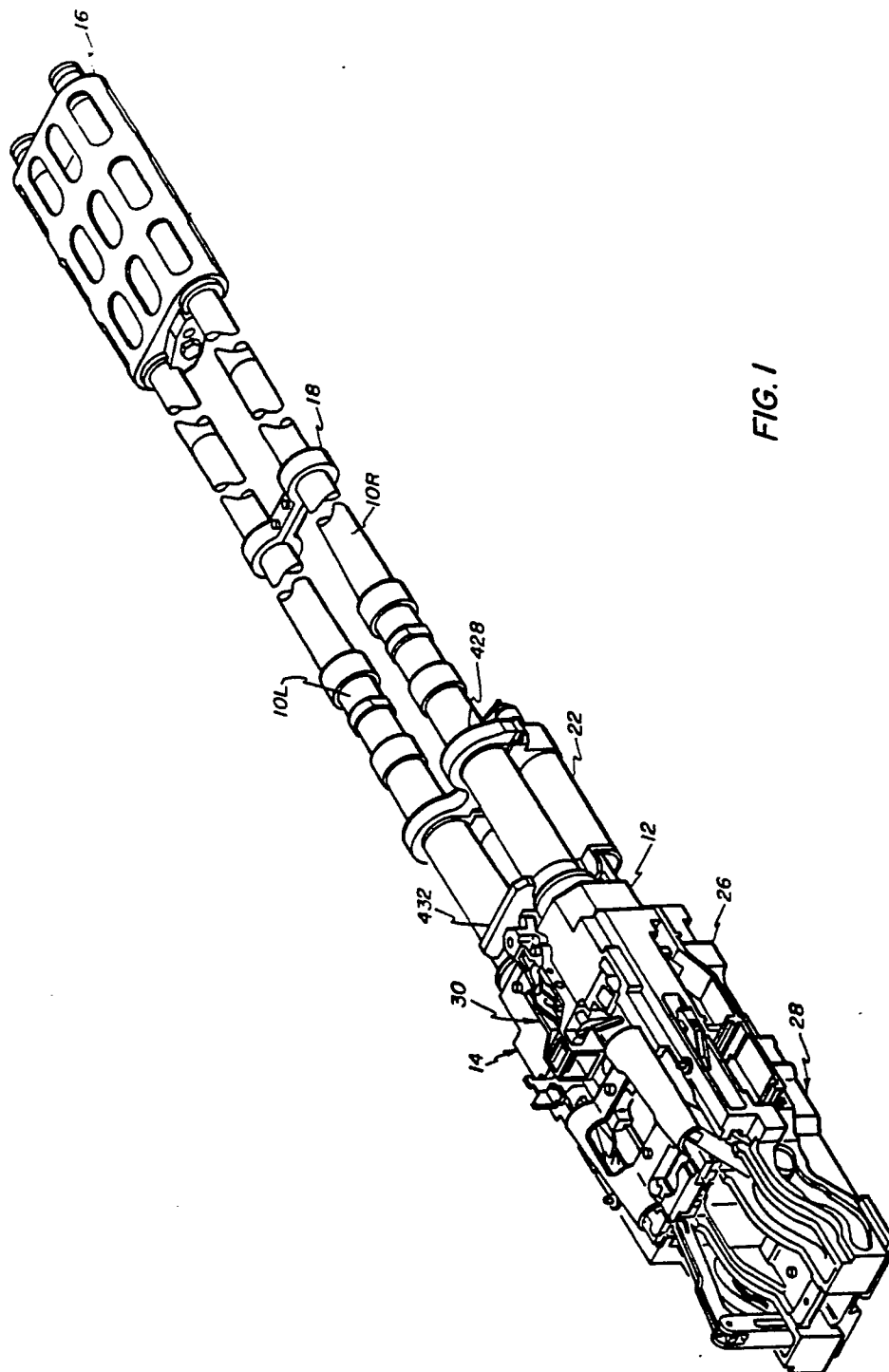
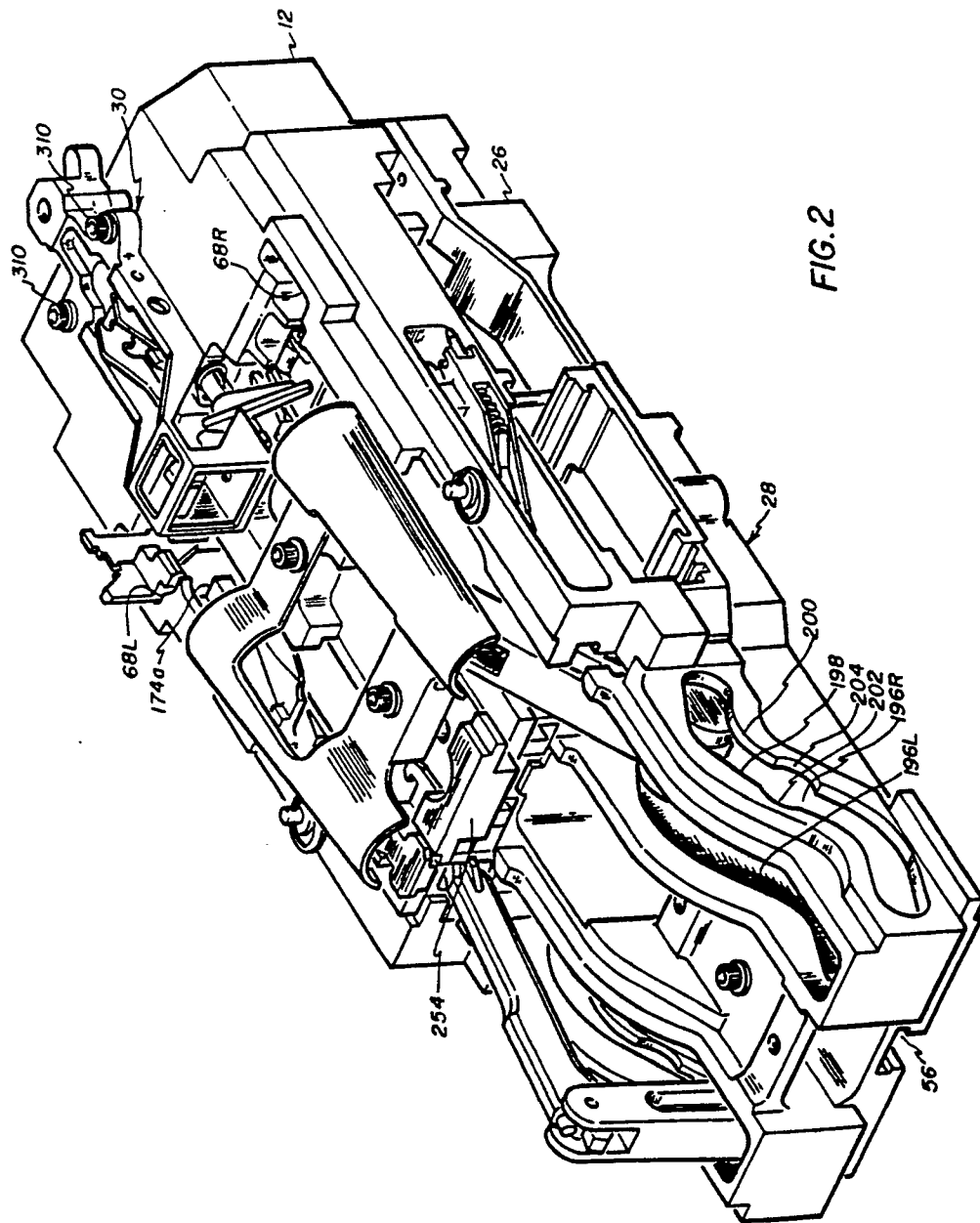
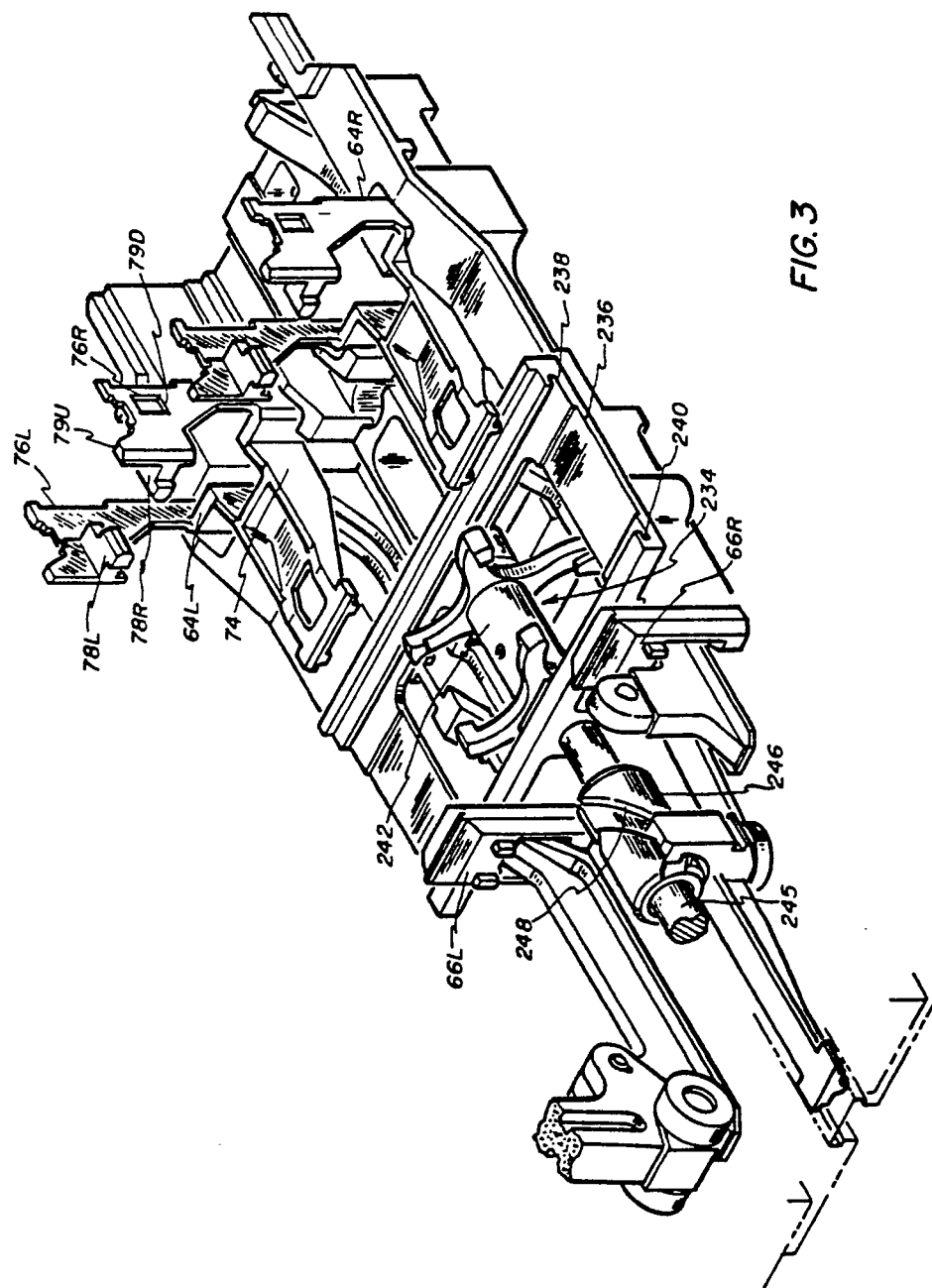


FIG. 1





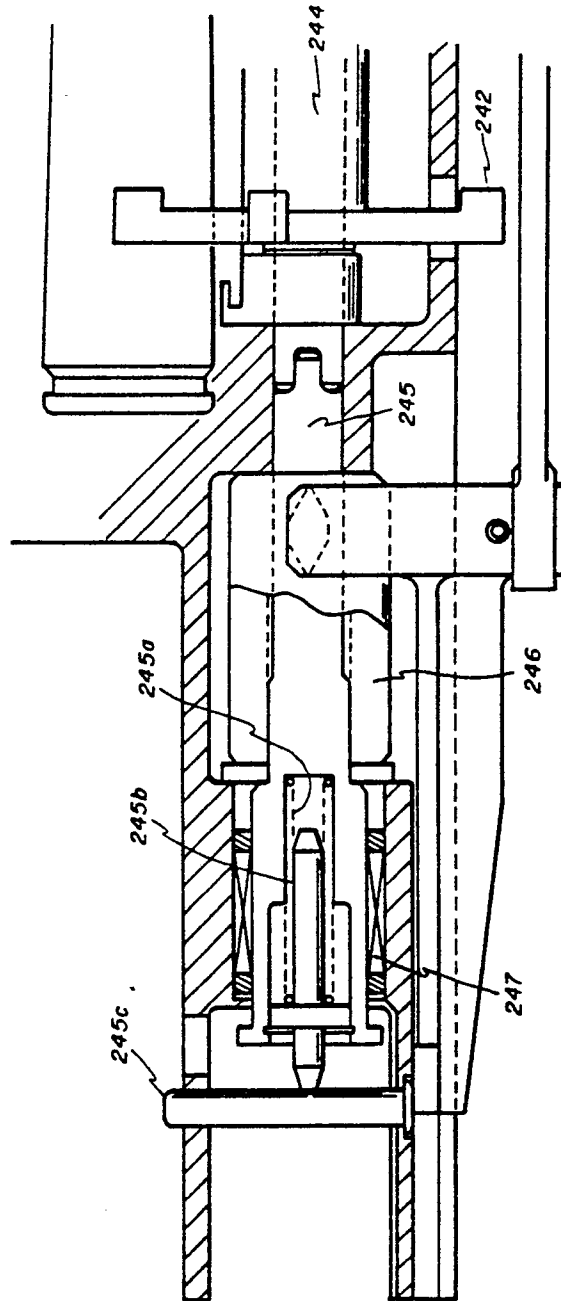


FIG. 3A

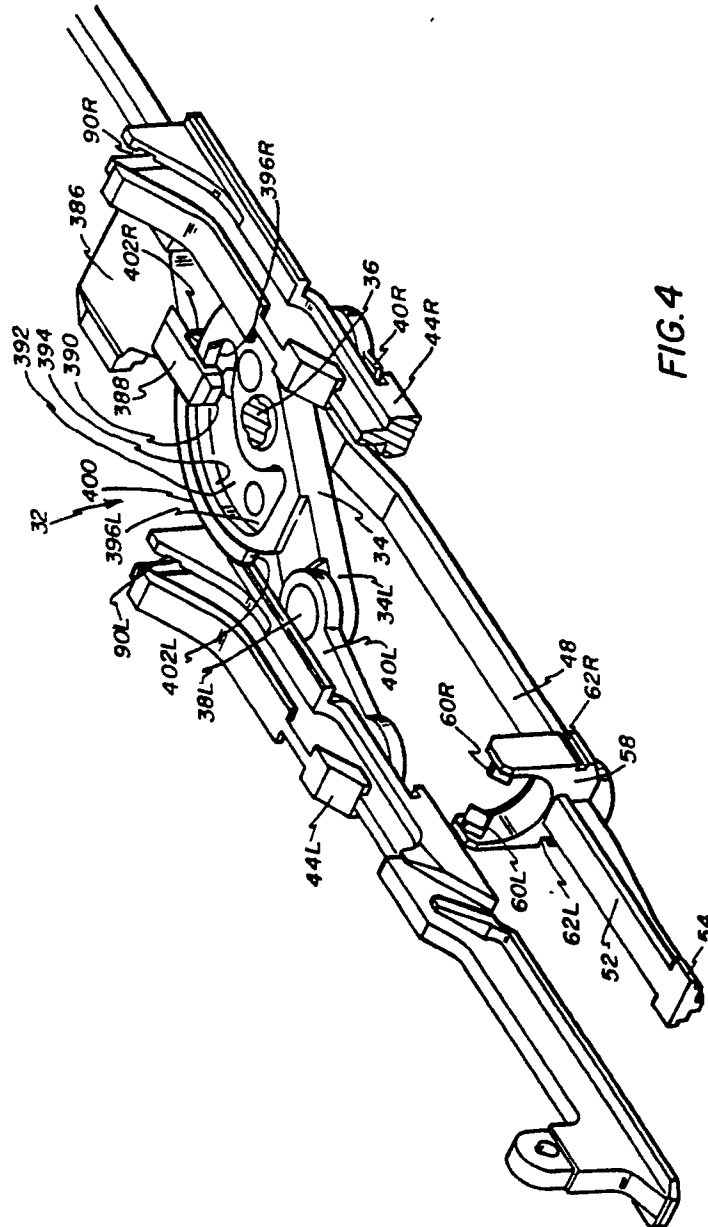


FIG. 4

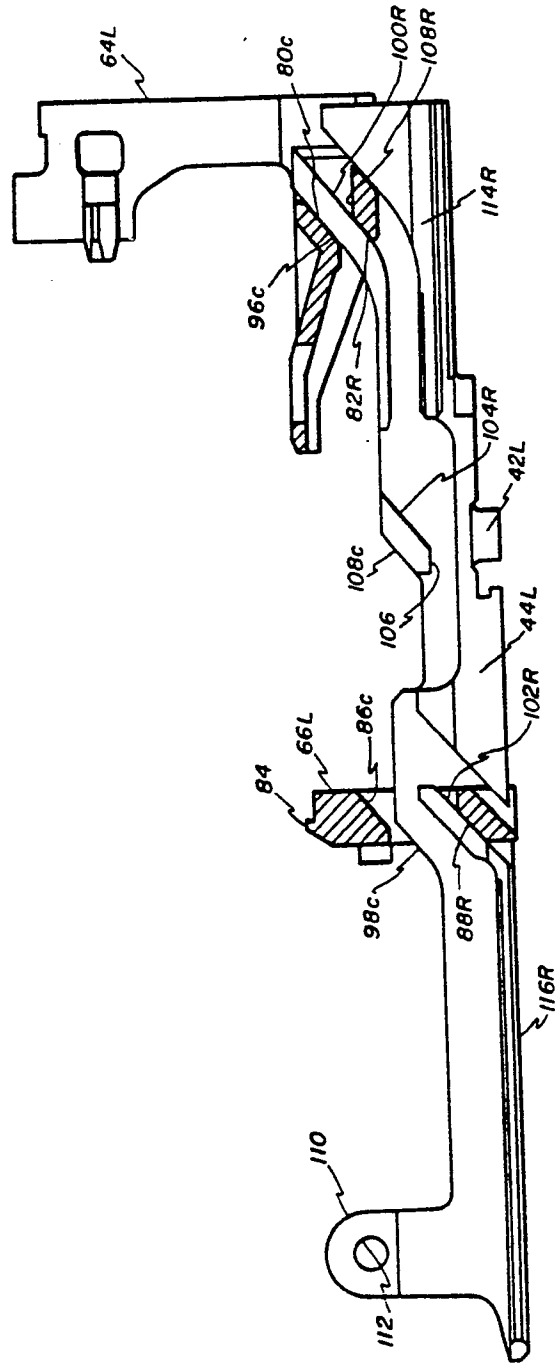
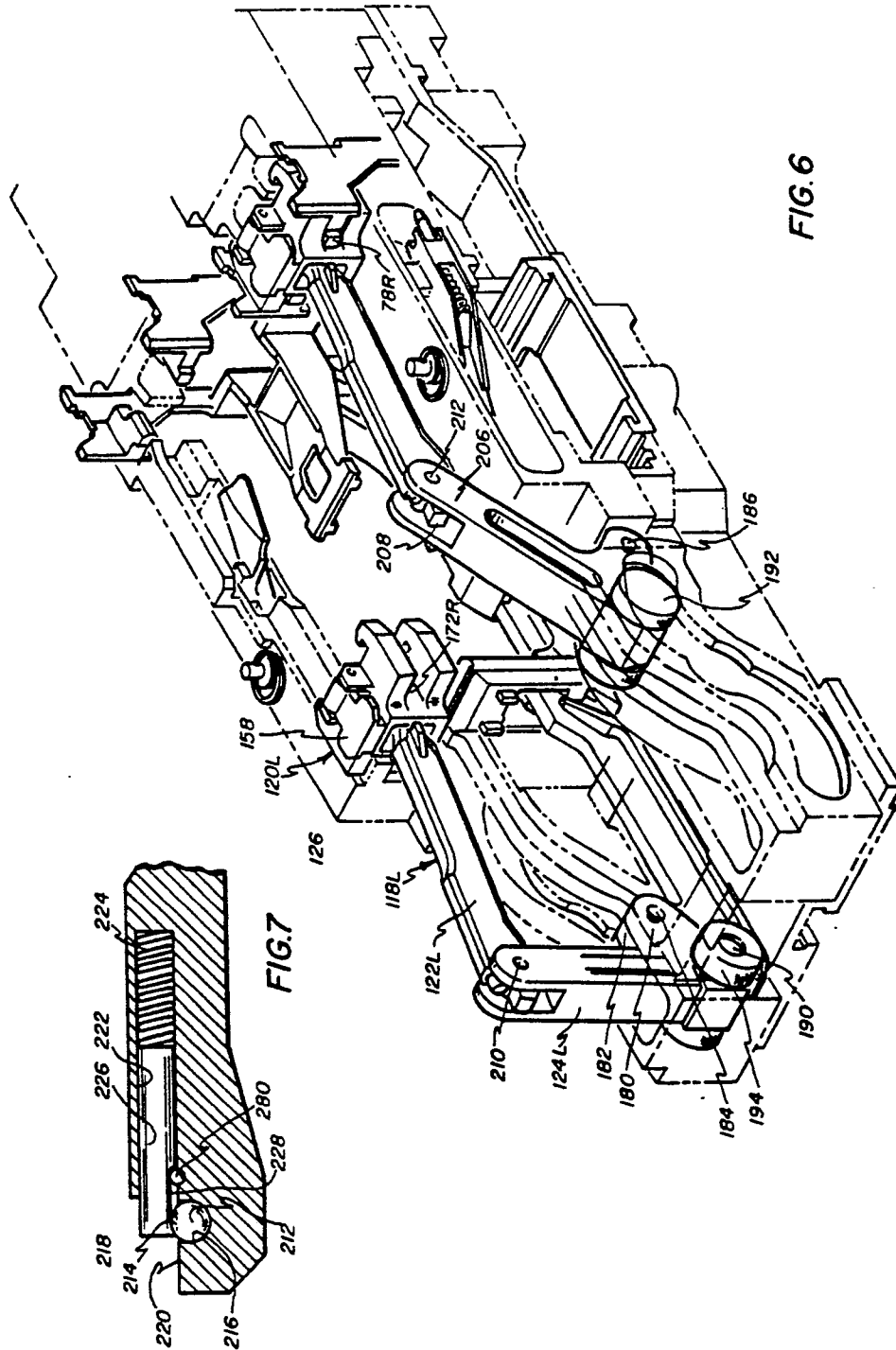


FIG. 5



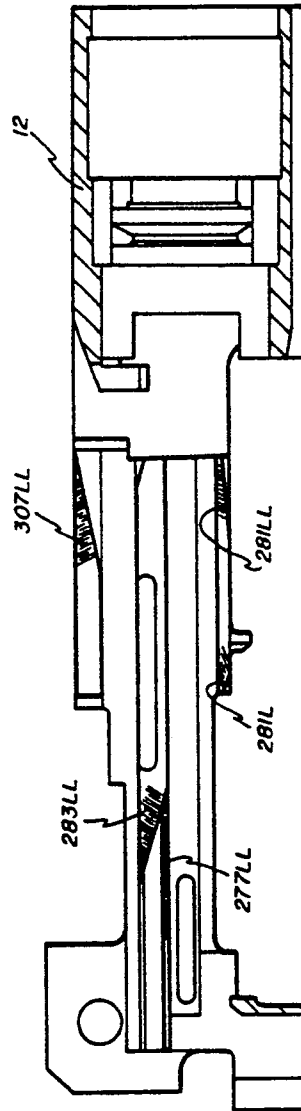


FIG. 6A

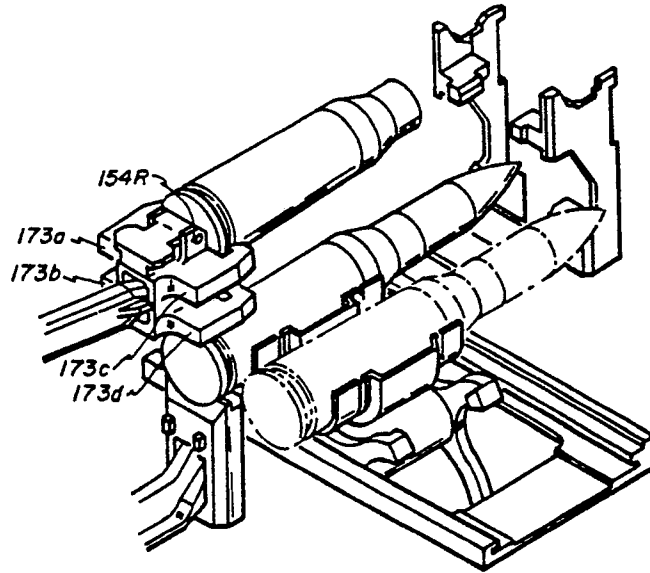


FIG. 10

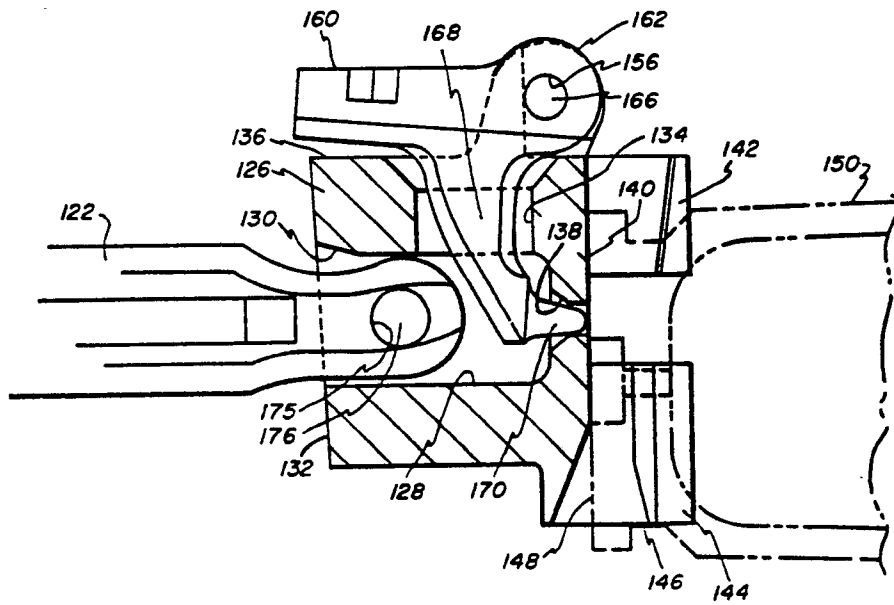
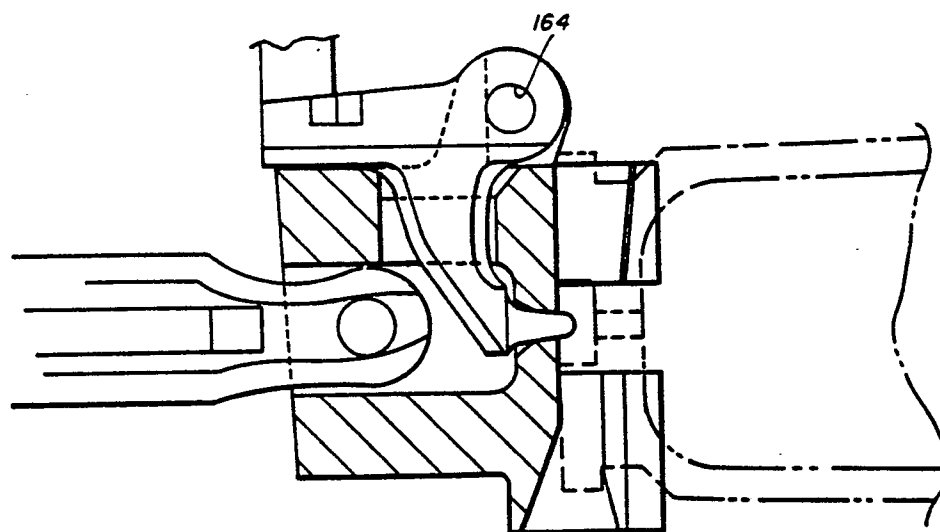
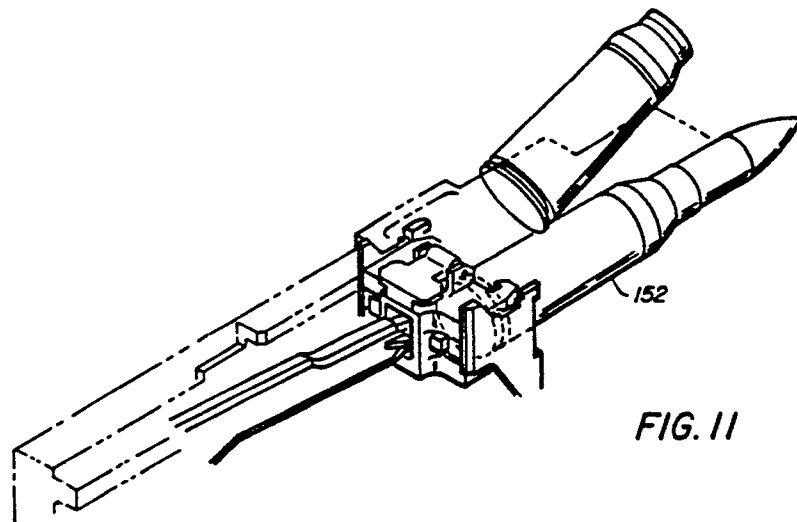


FIG. 8



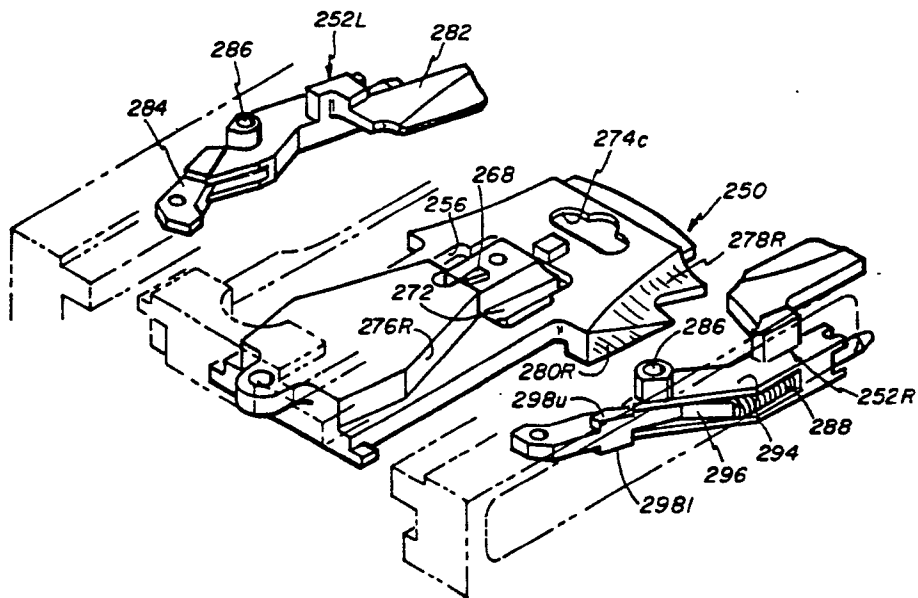


FIG. 12

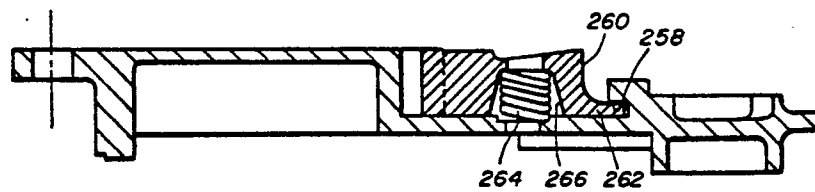


FIG. 12A

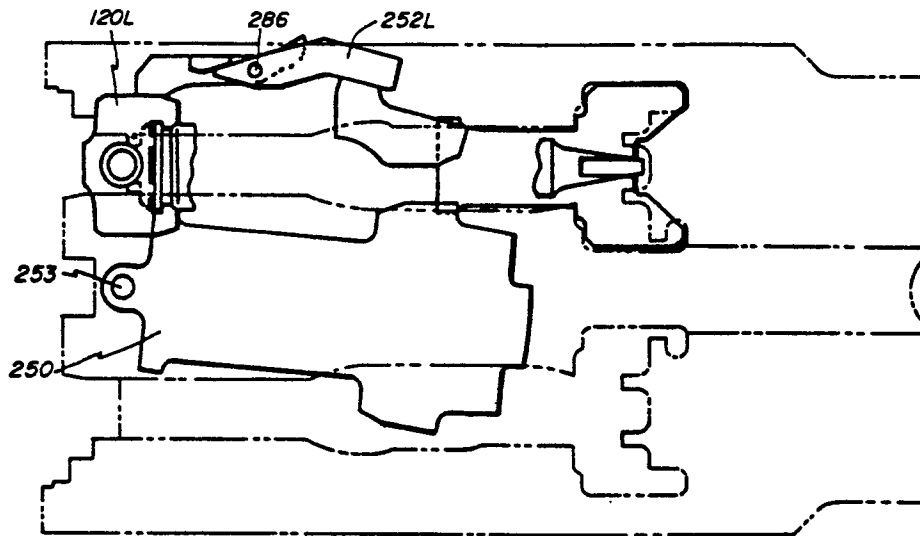


FIG. 13

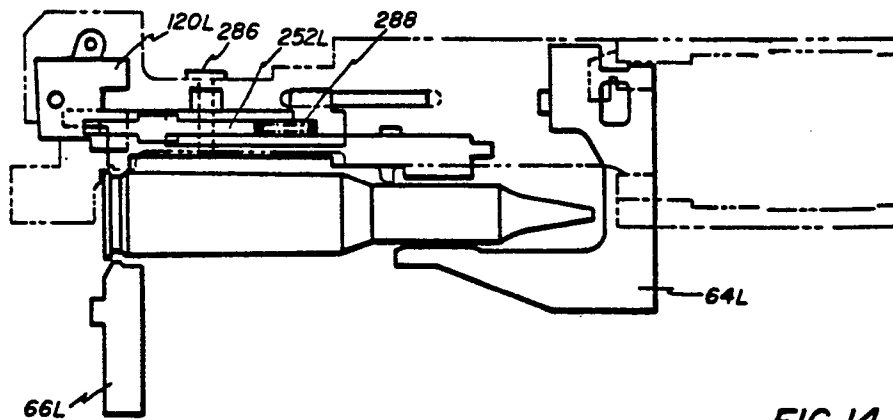


FIG. 14

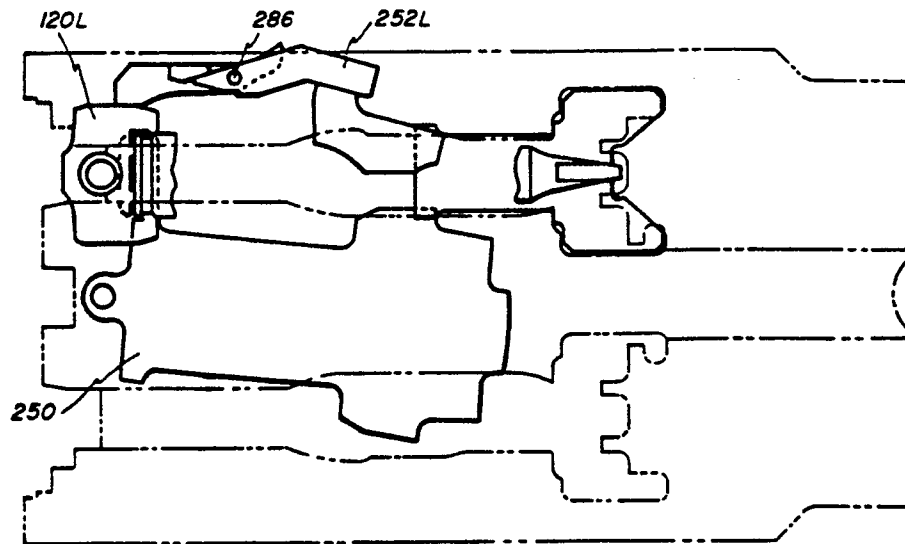


FIG. 15

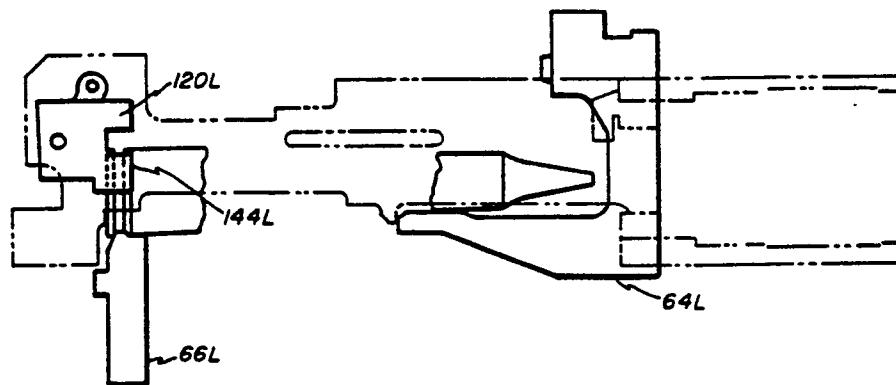


FIG. 16

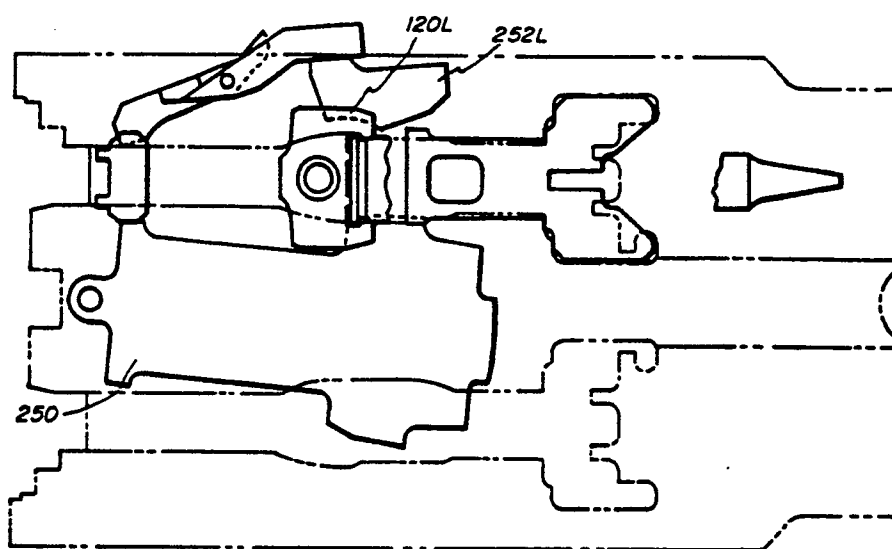


FIG. 17

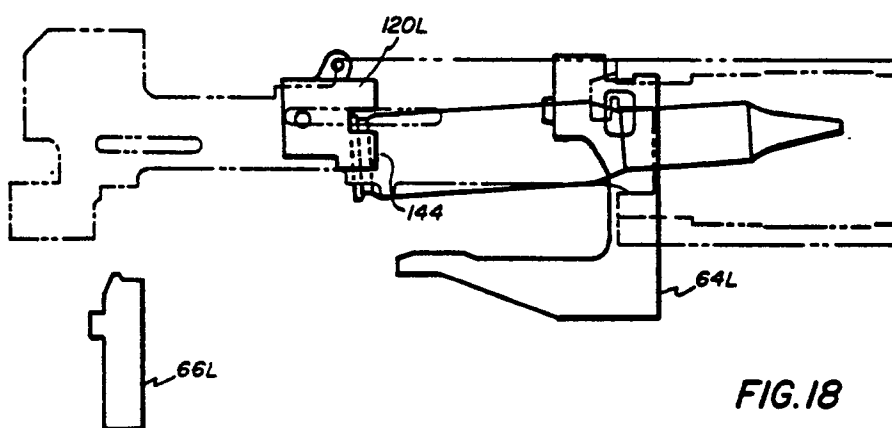


FIG. 18

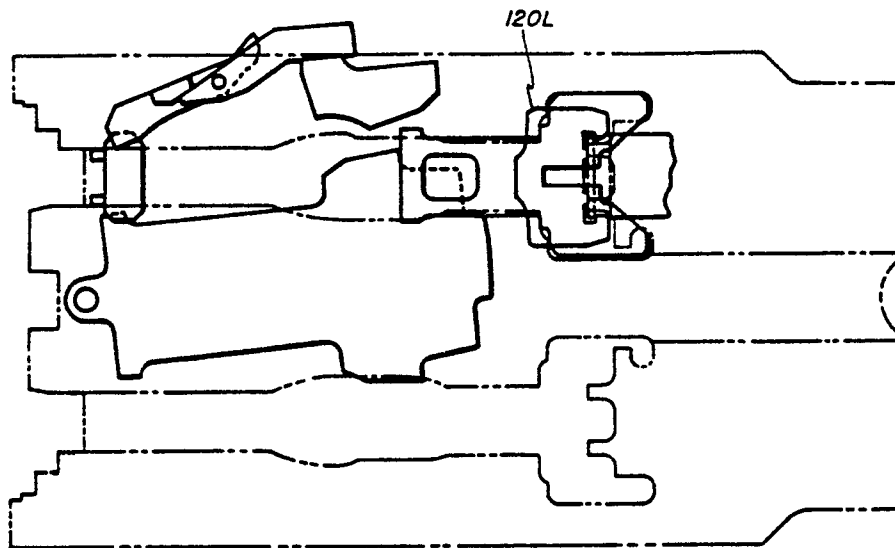


FIG. 19

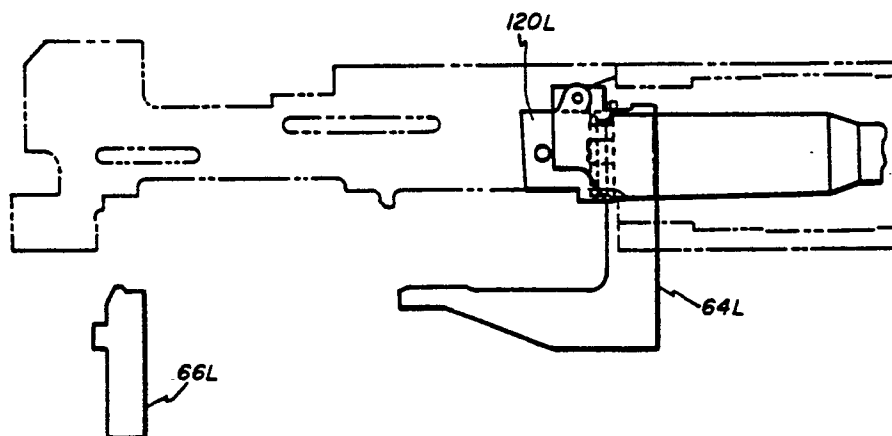


FIG. 20

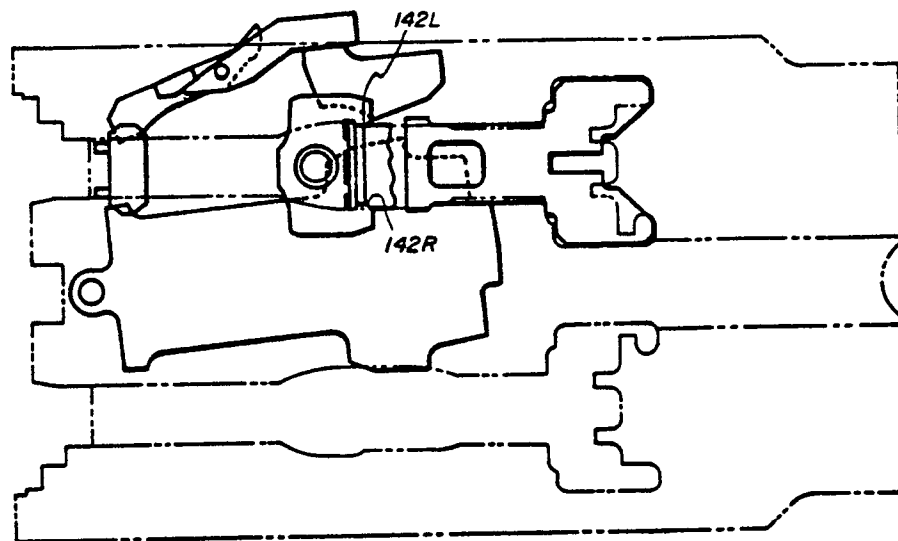


FIG. 21

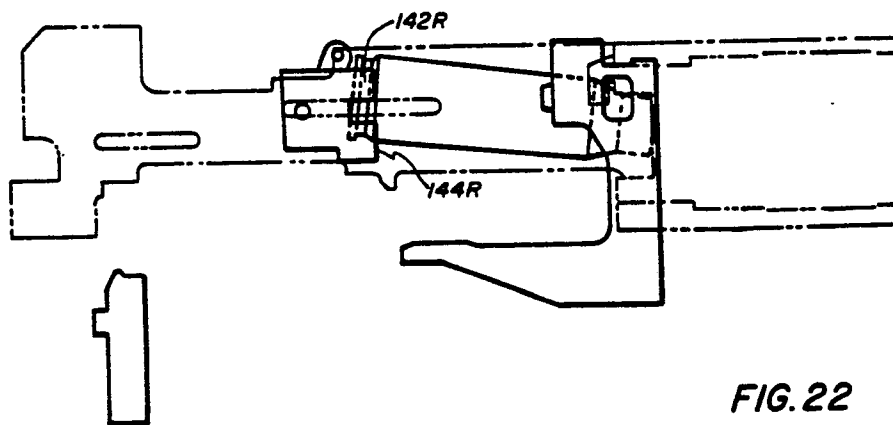


FIG. 22

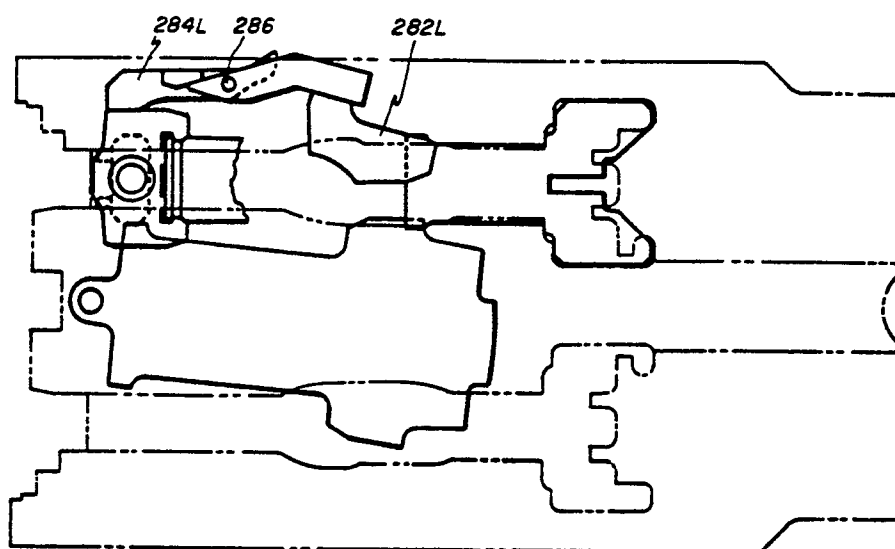


FIG. 23

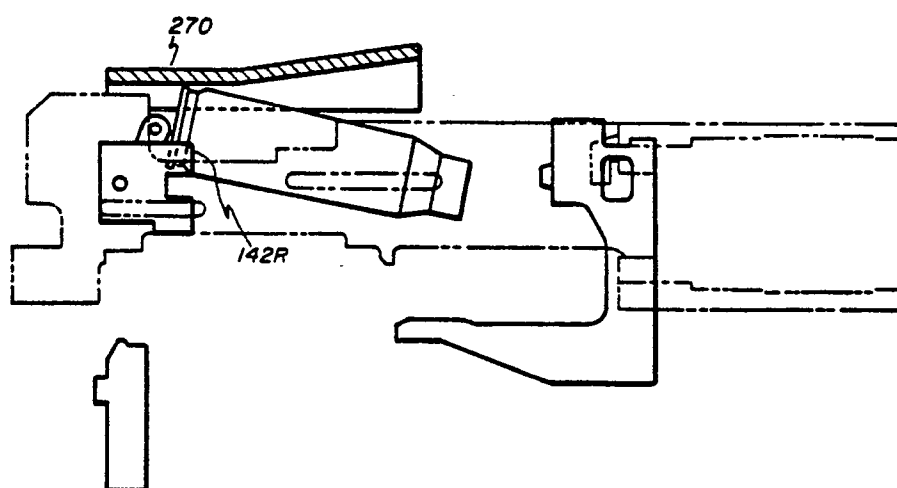


FIG. 24

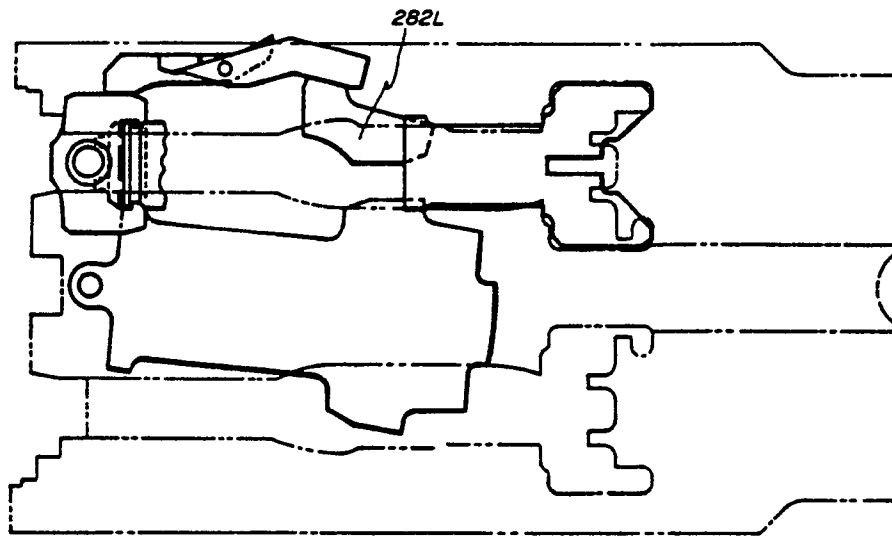


FIG. 25

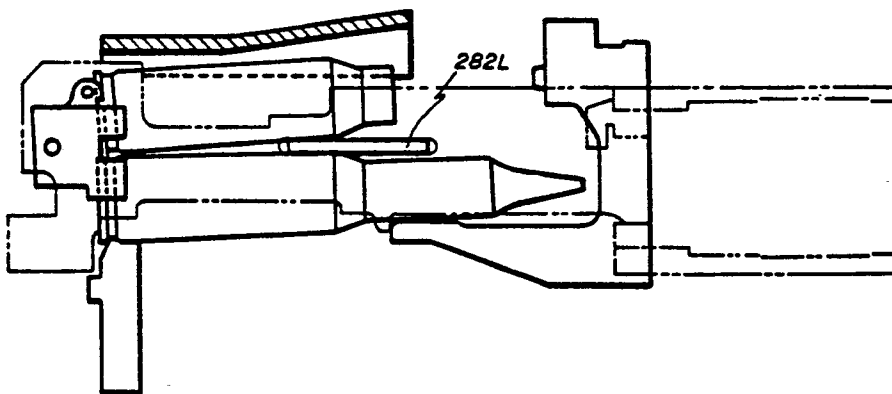


FIG. 26

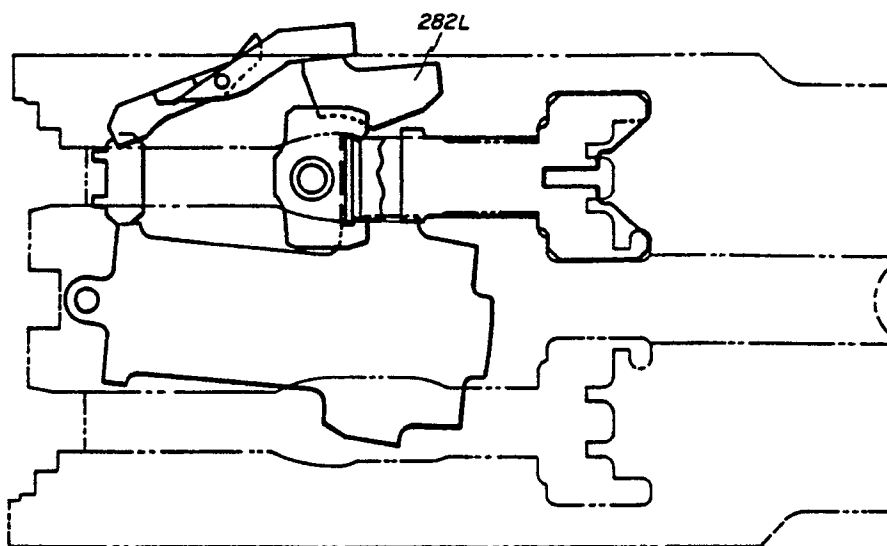


FIG. 27

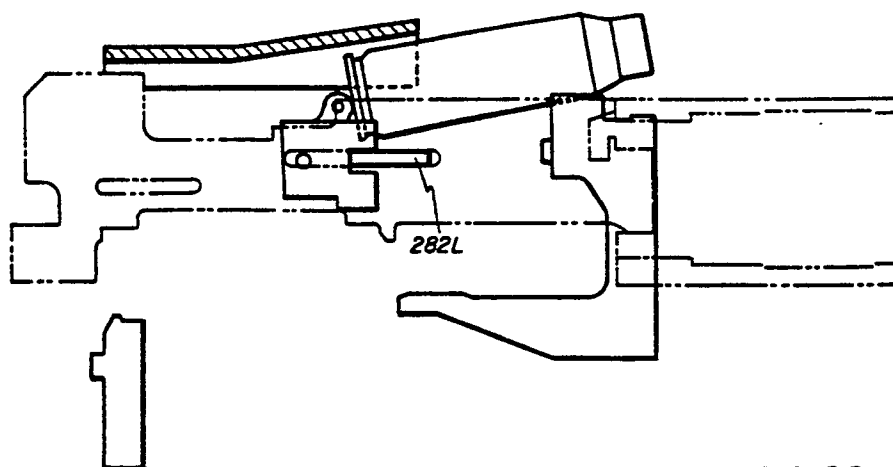


FIG. 28

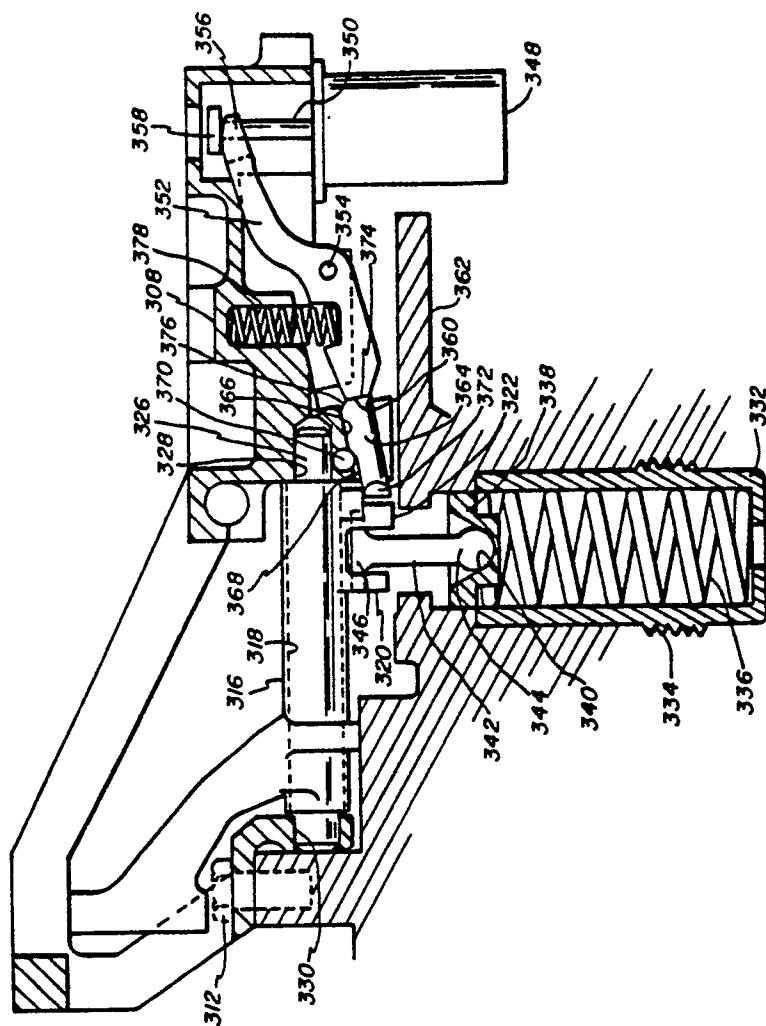


FIG. 29

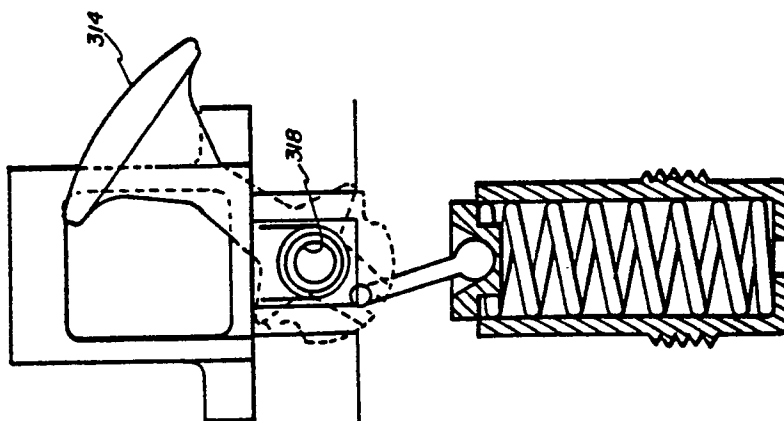


FIG. 30

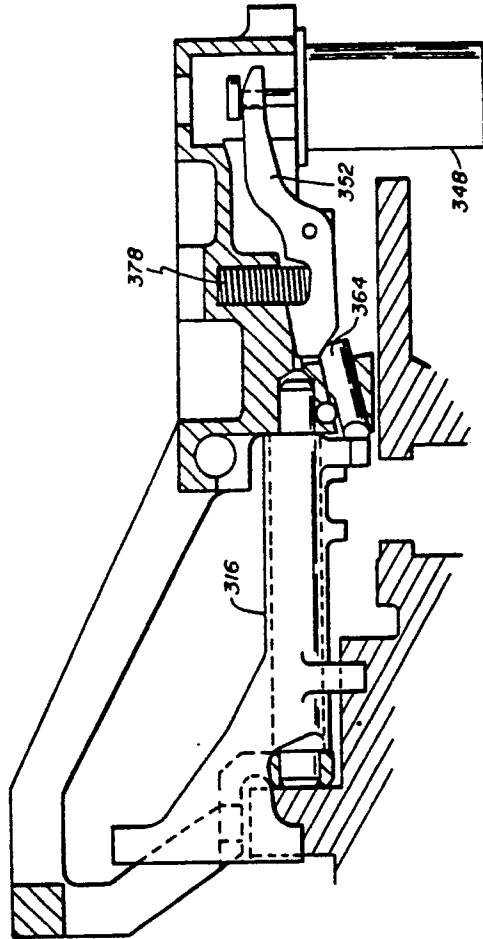


FIG. 31

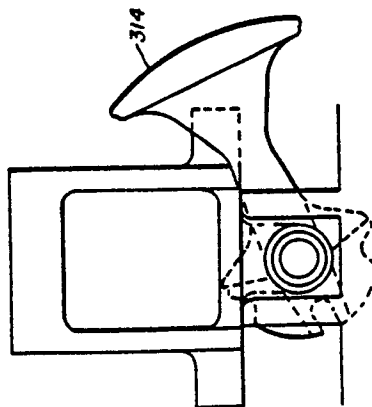


FIG. 32

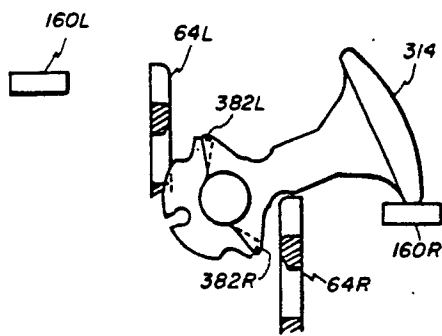


FIG. 33

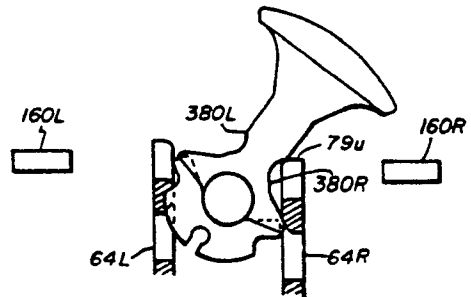


FIG. 34

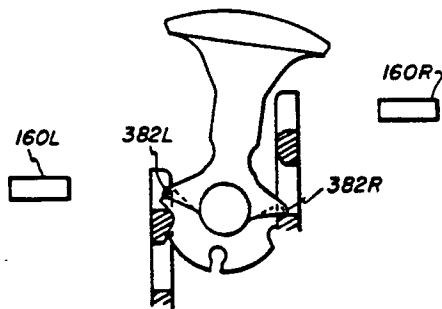


FIG. 35

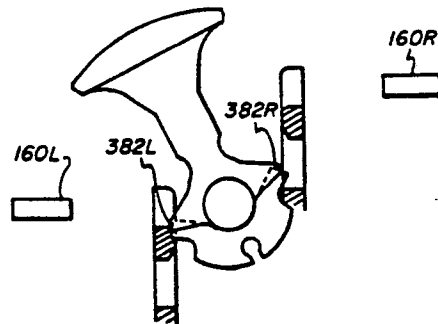


FIG. 36

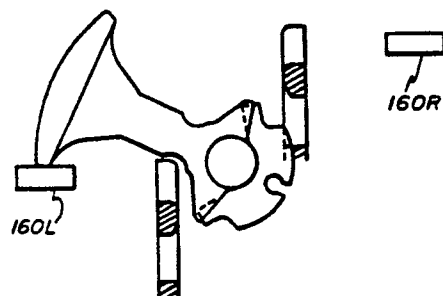


FIG. 37

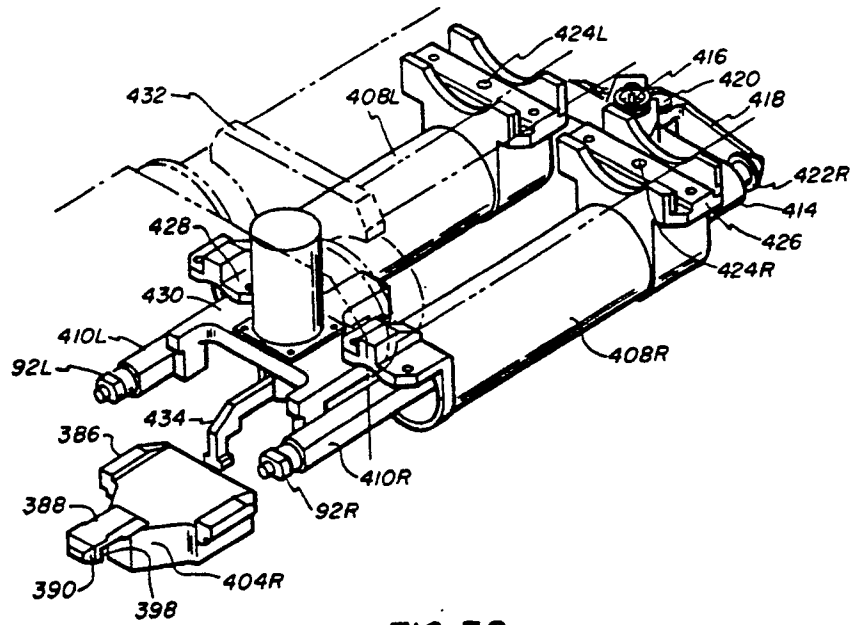


FIG. 38

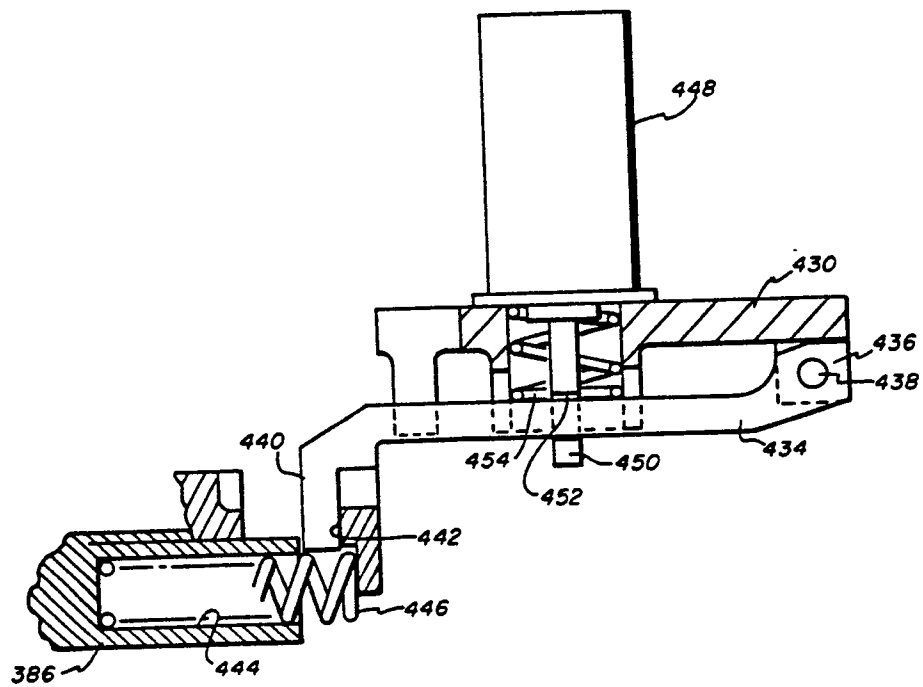


FIG. 39

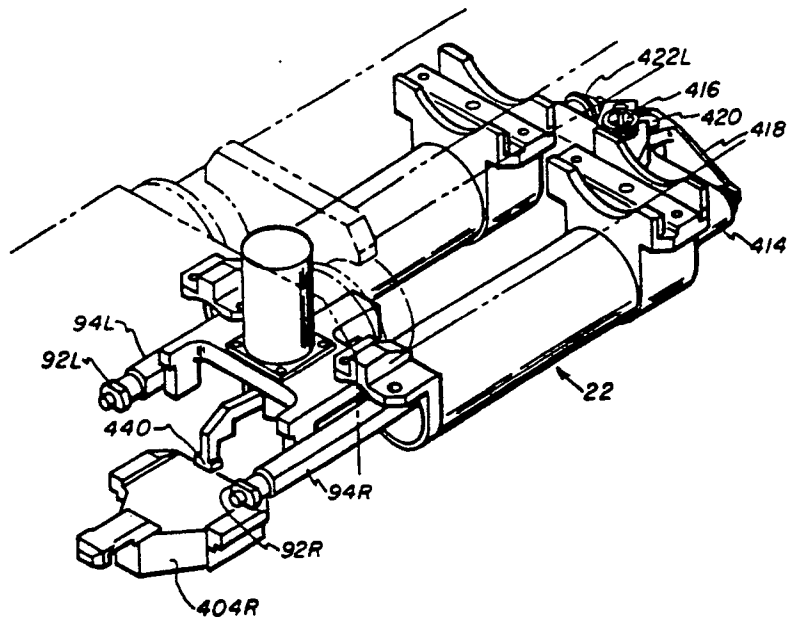


FIG. 40

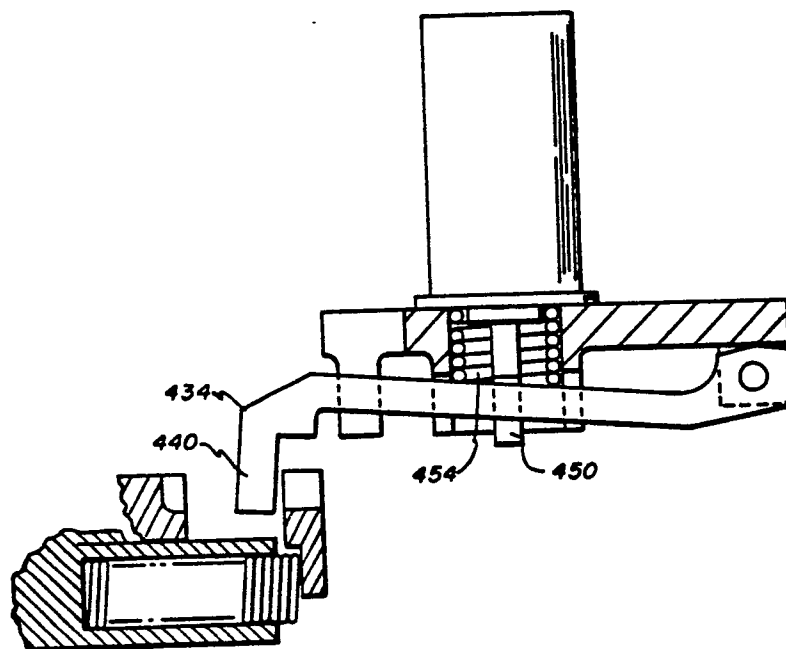
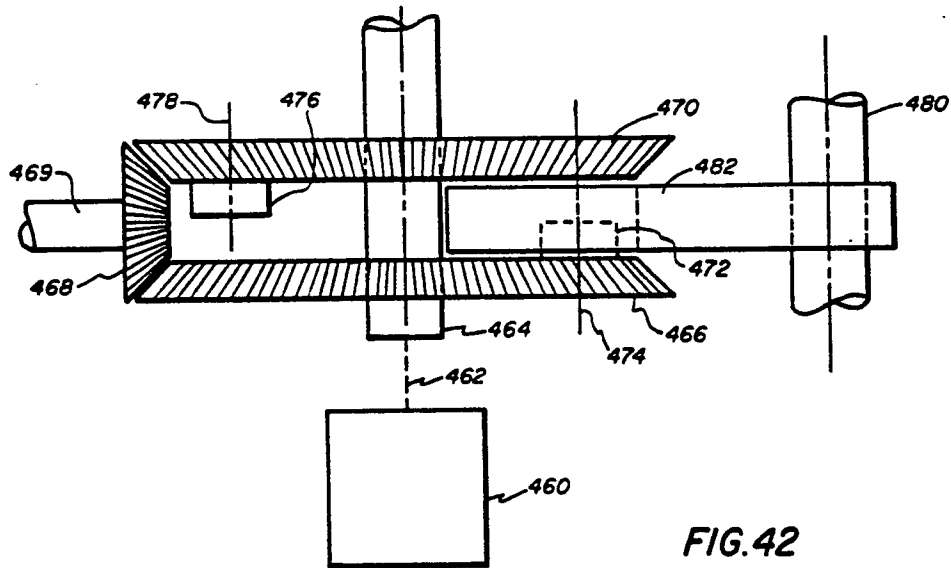
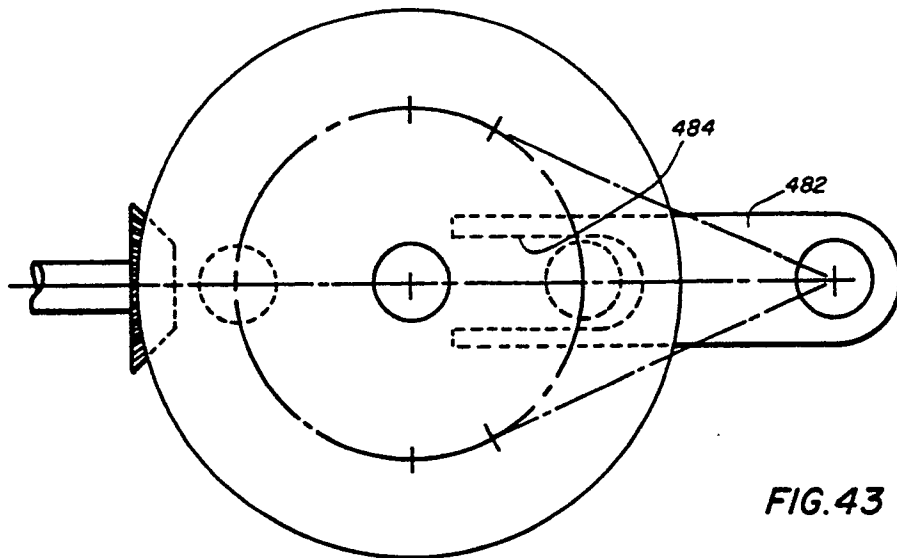


FIG. 41



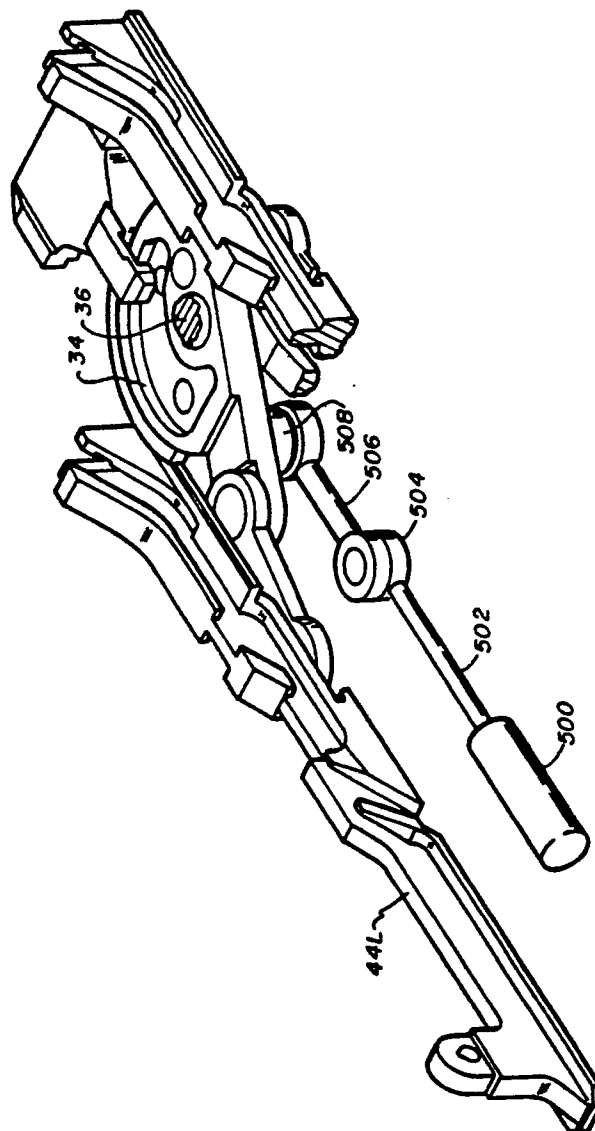


FIG. 44