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(43) **Pub. Date:** **Jun. 14, 2018**(54) **CONTROLLING CHILLED STATE OF A CARGO**(71) Applicants: **THERMO KING CORPORATION**,
Minneapolis, MN (US); **EMERSON CLIMATE TECHNOLOGIES - TRANSPORTATION SOLUTIONS APS**, St. Louis, MO (US)(72) Inventors: **Ole Thogersen**, Nyborg (DK); **Allan Dyrmoose**, Bogense (DK); **Dan Vad Steffensen**, Spentrup (DK)(21) Appl. No.: **15/820,976**(22) Filed: **Nov. 22, 2017****Related U.S. Application Data**

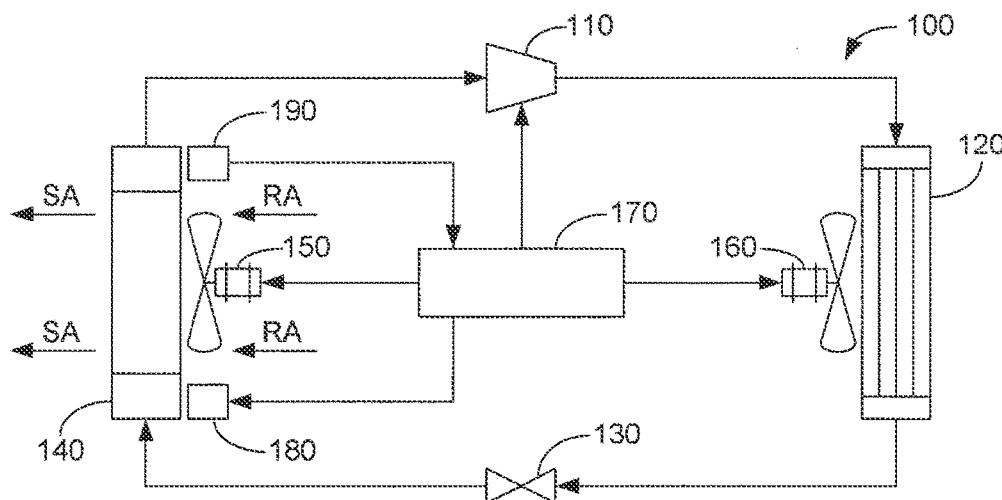
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(57) **ABSTRACT**

A method for operating a refrigeration system for a container for refrigerating chilled cargo includes providing a refrigeration system including a compressor and an evaporator fan associated with an evaporator. The method also includes determining the temperature of supply air and the temperature of return air. The method further includes determining one of a requirement for heating and a requirement for cooling based on the temperature of the return air and the temperature of the supply air. The method additionally includes activating the evaporator fan when a requirement for heating is determined and increasing the speed of the evaporator fan when increased heating is determined. The method also includes activating the compressor and the evaporator fan when a requirement for cooling is determined and increasing the power supplied to the compressor and maintaining the evaporator fan at a first speed when increased cooling is determined.



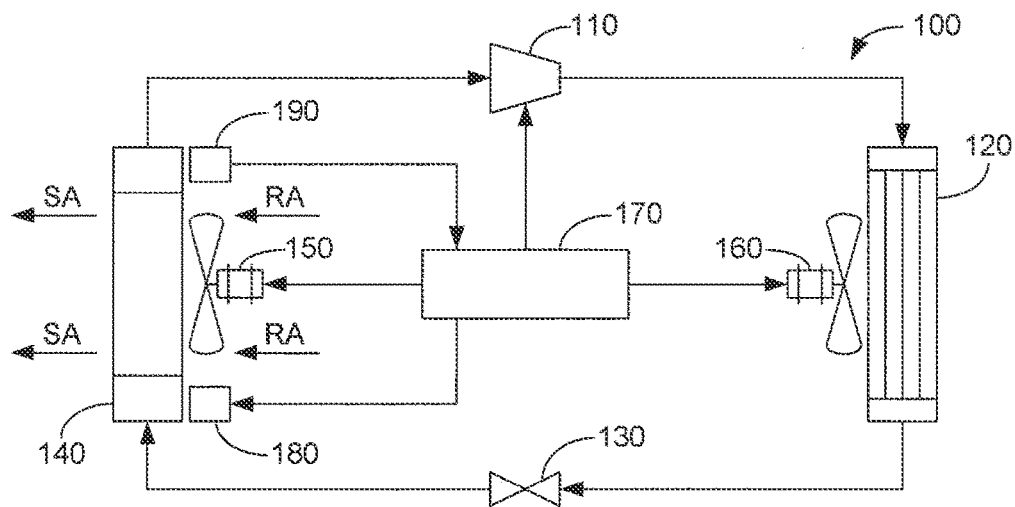


Fig. 1

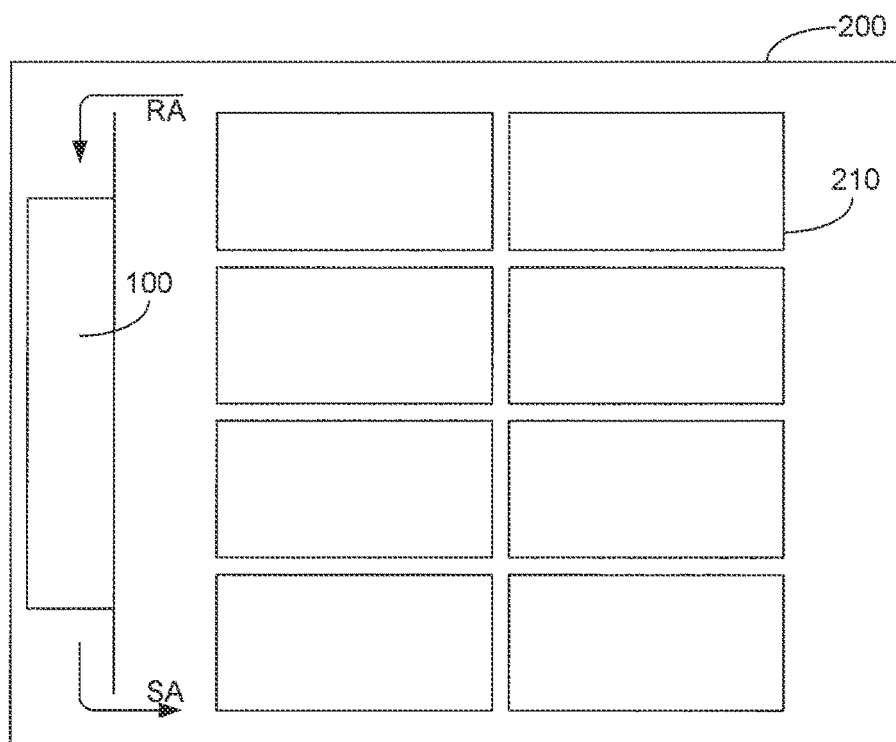
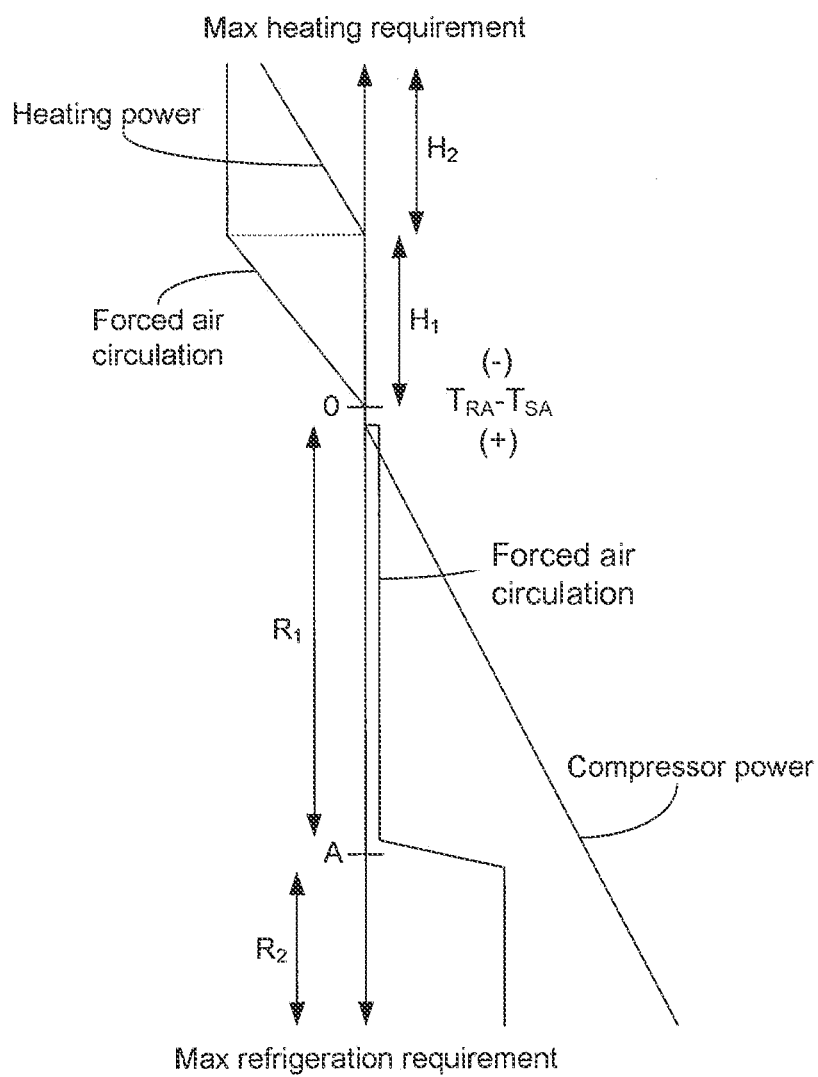


Fig. 2



CONTROLLING CHILLED STATE OF A CARGO

FIELD OF THE INVENTION

[0001] This invention relates to climate control in cargo containers and to devices for controlling the climate in cargo containers. In particular the invention relates to refrigeration systems for use in cargo containers and methods for operating such systems for controlling a cargo in a chilled state.

BACKGROUND OF THE INVENTION

[0002] The present invention relates to transporting and storing temperature sensitive cargo over long periods of time using a controlled climate in the space where the cargo is loaded. Climate control includes controlling the temperature of the cargo within a certain acceptable range. Controlling the temperature includes bringing the temperature of the cargo into the acceptable range (by refrigerating or heating) and maintaining the temperature within that range. Climate control may also include controlling other parameters such as humidity and composition of the atmosphere.

[0003] Refrigeration is the process of removing heat from an enclosed space, or from a substance, and moving it to a place where it is unobjectionable. The primary purpose of refrigeration is lowering the temperature of the enclosed space or substance and then maintaining that lower temperature.

[0004] One commonly used refrigeration technique is the vapor-compression cycle. The vapor-compression cycle is used in most household refrigerators as well as in many large commercial and industrial refrigeration systems.

[0005] A refrigerated container or reefer is a shipping container used in intermodal freight transport, including rail, ship and truck, where the cargo is refrigerated (chilled or frozen) for the transportation of temperature sensitive cargo. A reefer will usually have an integral refrigeration unit.

[0006] The reliability of the refrigeration unit is of paramount importance. The temperature of temperature sensitive cargo should be kept within predefined limits. Some cargo must be maintained frozen, and the temperature of any part of the frozen cargo must be kept below a predefined freezing temperature which depends on the cargo, e.g. below -18 degrees C. or lower, while other cargo, in particular commodities such as fresh meat, fresh fruit and vegetables, should be kept chilled to stay fresh, but not frozen. For chilled fruit and vegetables there is a lowest acceptable temperature below which the commodity will begin degrading and lose its freshness. Such temperature is dependent upon the type of fruit.

SUMMARY OF THE INVENTION

[0007] In one embodiment, the invention provides a method and a system for controlling the temperature of a cargo in a chilled state. In chilled state the cargo is to be maintained at or near a set-point temperature T_{SP} and for certain commodities and not below the set-point temperature. Whether heating or refrigeration is required the invention ensures a minimal consumption of energy with proper use of forced air circulation, refrigeration and heating depending on the actual requirement.

[0008] In one embodiment, the invention provides a method for operating a refrigeration system for a container for refrigerating chilled cargo. The method includes provid-

ing a refrigeration system including a compressor, a condenser, and an evaporator connected in series, an evaporator fan associated with the evaporator, and a heater. The method also includes determining the temperature of the supply air discharged into the container and the return air from the container, and determining one of a requirement for heating and a requirement for cooling based on the temperature of the return air and the temperature of the supply air. The method further includes activating the evaporator fan when a requirement for heating is determined and increasing the speed of the evaporator fan when increased heating is determined, and activating the compressor and the evaporator fan when a requirement for cooling is determined and increasing the power supplied to the compressor and maintaining the evaporator fan at a first speed when increased cooling is determined.

[0009] Another embodiment of the invention is directed to a refrigeration system for a container for refrigerating chilled cargo. The system includes a compressor, a condenser, and an evaporator connected in series. The system also includes a heater and an evaporator fan associated with the evaporator, where the evaporator fan is operable to discharge supply air to the container and to receive return air from the container. The system further includes sensors configured to sense the temperature of the supply air and the temperature of the return air. The controller is programmed to determine one of a requirement for heating and a requirement for cooling based on the temperature of the return air and the temperature of the supply air. Where the controller is further programmed to activate the evaporator fan when a requirement for heating is determined and to increase the speed of the evaporator fan when increased heating is determined. Where the controller is programmed to activate the compressor and the evaporator fan when a requirement for cooling is determined and to increase the power supplied to the compressor and maintain the evaporator fan at a first speed when increased cooling is determined.

[0010] Other aspects of the invention will become apparent by consideration of the detailed description and accompanying drawings.

BRIEF DESCRIPTION OF THE DRAWINGS

[0011] FIG. 1 shows schematically a refrigeration system according to the invention.

[0012] FIG. 2 shows a refrigerated container with the refrigeration system in FIG. 1 installed.

[0013] FIG. 3 is a diagram illustrating the operation of the invention in dependence on the actual requirement for heating or refrigeration.

DETAILED DESCRIPTION OF THE INVENTION

[0014] Before any embodiments of the invention are explained in detail, it is to be understood that the invention is not limited in its application to the details of construction and the arrangement of components set forth in the following description or illustrated in the following drawings. The invention is capable of other embodiments and of being practiced or of being carried out in various ways.

[0015] FIG. 1 is a simplified diagram of the basic components of a typical one-stage vapor-compression refrigeration system 100 according to the invention. In this cycle, a circulating refrigerant enters the compressor 110 as a vapor.

In the compressor the vapor is compressed and exits the compressor superheated. The superheated vapor travels through the condenser **120** which first cools and removes the superheat and then condenses the vapor into a liquid by removing additional heat at constant pressure and temperature. The liquid refrigerant goes through an expansion valve **130** (also called a throttle valve) where its pressure abruptly decreases, causing flash evaporation and auto-refrigeration of, typically, less than half of the liquid. That results in a mixture of liquid and vapor at a lower temperature and pressure. The cold liquid-vapor mixture then travels through the evaporator **140** coil or tubes and is completely vaporized by cooling the warm return air RA returning from the refrigerated space being blown by an evaporator fan **150** across the evaporator coil or tubes. The cool supply air SA is blown into the refrigerated space. The resulting refrigerant vapor returns to the compressor inlet to complete the thermodynamic cycle. A condenser fan **160** removes condensation heat from the condenser **120**. A controller **170** controls the operation of the refrigeration system and its individual components.

[0016] During operation water vapor will condensate on the evaporator **140** and form a layer of ice which will degrade the efficiency of the evaporator. The ice is removed in defrosting cycles where the compressor **110** and the evaporator fan **150** are inactivated, and a heater **180** is activated which will heat the evaporator **140**. A temperature sensor **190** senses the temperature of the evaporator **140** and when it has been determined, based on the sensed evaporator temperature, that the ice is melted, the compressor **110** is again activated. When the temperature of the evaporator is sufficiently low the evaporator fan **150** is activated and the refrigeration system is in operation again.

[0017] The refrigeration system **100** can have one or more evaporator fans **150**. The power of the evaporator fan motors can be controlled in two or more steps or continuously by the controller **170**. For simplicity, only high speed operation and low speed operation are described, but the person having ordinary skill in the art will understand that the described method applies in general to motors with controllable speed.

[0018] FIG. 2 shows schematically a portion of a refrigerated container **200** loaded with cargo **210** to be refrigerated. The container **200** has a refrigeration system **100** installed in one end, and the container has doors (not shown) in the opposite end for loading and unloading the cargo **210**. The evaporator fan or fans **150** of the refrigeration system **100** blow refrigerated supply air SA into the container where it circulates around the cargo **210** and returns as return air RA to the refrigeration system **100**.

[0019] The energy required for circulating the air in the container is ultimately dissipated as heat in the container due to friction. Depending on whether the evaporator fan **150** is operated in a low speed mode or in a high speed mode it delivers from a few hundred watts up to a few kilowatts (kW) which is dissipated as heat in the container. This energy adds to the energy that enters the container from the ambient and the heat that is generated by the cargo itself, all of which must be removed by the refrigeration system. Assuming efficiencies of 100% of both the evaporator fan and the refrigeration system, for each kW consumed by the evaporator fan another kW will be consumed by the refrigeration system.

[0020] FIG. 3 illustrates the operation of the invention in dependence on the actual requirement for heating or refrigeration.

Based on observed temperature T_{RA} of the return air RA and the temperature T_{SA} of the supply air SA and the difference $T_{RA} - T_{SA}$ of the two temperatures the requirement for heating or refrigeration is calculated.

[0021] When the observed temperature difference $T_{RA} - T_{SA}$ indicates that neither heating nor refrigeration is required, then none of the compressor, the evaporator fan and the heater is operated, since there is nothing to correct. However, at predetermined intervals the evaporator fan is activated to circulate air in the container and to draw the return air stream past the return air temperature sensor to measure its temperature to determine whether heating or refrigeration is needed.

[0022] When a requirement for moderate heating is determined as in interval H_1 , the evaporator fan is activated to circulate air in the container at a rate where the friction heat generated by the air flow satisfies the need for heating. This is possible with fan motors with continuously variable speed, and with other motors it can be obtained by pulse width modulation (PWM) of the electric power supplied to the motors. The speed of other (traditional) fan motors can be controlled by turning them on and off at relatively longer intervals resulting in the correct mean value of the motor speed.

[0023] At higher requirements for heating than satisfied by the evaporator fan alone, as in interval H_2 , the evaporator fan is operated at its full capacity and supplemented by the heater **180**. The heater power is adjusted so that the friction heat from the air flow plus the heat generated by the heater satisfy the need for heating. The electric power supplied to the heater can be varied e.g. by pulse width modulating the power.

[0024] When a requirement for moderate refrigeration is determined as in interval RI, the compressor **110** is activated and the evaporator fan is activated to circulate the air in the container and pass it through the evaporator coil to be refrigerated. The circulation of air results in friction heat dissipated in the container which adds to the energy to be removed by refrigeration. The evaporator fan motor is therefore operated at a low speed which is sufficient to circulate the air so as to meet the requirement for refrigeration and to dissipate as little friction heat as possible. Variations in the refrigeration requirement are accommodated for by regulating the compressor power.

[0025] Letter A in FIG. 3 indicates a requirement for refrigeration determined by the set-point temperature T_{SP} , by the ambient conditions outside of the container, which leaks heat energy into the container, and by the heat energy generated by the cargo, all of which are known or can be determined by measurements or observations or possibly estimated. At requirements for refrigeration higher than the requirement value A it is necessary to circulate the air in the container at a high rate and correspondingly operate the evaporator fan at a high speed. Variations in the refrigeration requirement are accommodated for by regulating the compressor power.

[0026] The change in fan speed at requirement A affects the air flow through the evaporator, and therefore the compressor power is adjusted accordingly.

[0027] The interval around the requirement value A is relatively narrow and the speed of the evaporator fan motor may be varied continuously over this interval.

[0028] Various features and advantages of the invention are set forth in the following claims.

1-11. (canceled)

12. A refrigeration system for use in transport, the refrigeration system comprising:

- a compressor, a condenser, and an evaporator connected in series;
- an evaporator fan associated with the evaporator and configured to discharge supply air from the refrigeration system to a refrigerated space;
- a supply air temperature sensor configured to sense a supply air temperature of the supply air;
- a return air temperature sensor configured to sense a return air temperature of return air returning from the refrigerated space to the refrigeration system; and
- a controller programmed to determine a requirement for moderate heating and a requirement for increased heating based on the supply air temperature and the return air temperature, to activate the evaporator fan when the requirement for moderate heating is determined, and to increase a speed of the evaporator fan when the requirement for increased heating is determined.

13. The refrigeration system of claim 12, further comprising a heater configured to heat the supply air, wherein the controller is programmed to activate the heater to heat the supply air when the requirement for increased heating is determined.

14. The refrigeration system of claim 13, wherein the evaporator fan is operated at a first speed to circulate air in the refrigerated space and the heater is adjusted so that a friction heat from air circulated by the evaporator fan in the refrigerated space plus heat generated by the heater satisfies the requirement for increased heating.

15. The refrigeration system of claim 12, wherein the controller is programmed to activate the evaporator fan to circulate air in the refrigerated space at a rate where a friction heat generated by the circulated air satisfies the requirement for moderate heating.

16. The refrigeration system of claim 12, wherein the controller is programmed to determine the requirement for moderate heating and the requirement for increased heating at predetermined time intervals.

17. The refrigeration system of claim 12, wherein the controller is programmed to determine the requirement for moderate heating and the requirement for increased heating based on a difference between the return air temperature and the supply air temperature.

18. The refrigeration system of claim 12, wherein the controller is programmed to determine a requirement for cooling based on the supply air temperature and the return air temperature, and to activate the compressor and the evaporator fan when the requirement for cooling is determined.

19. The refrigeration system of claim 18, wherein the controller is programmed to increase a power required by the compressor when increased cooling is determined.

20. The refrigeration system of claim 12, wherein the controller is programmed to deactivate the compressor and the evaporator fan when neither heating nor refrigeration is required.

21. A method for operating a refrigeration system for use in transport, the method comprising:

- determining a supply air temperature of supply air discharged by the refrigeration system to a refrigerated space;
- determining a return air temperature of return air returning from the refrigerated space to the refrigeration system;
- determining a requirement for moderate heating and a requirement for increased heating based on the supply air temperature and the return air temperature;
- activating an evaporator fan of the refrigeration system when the requirement for moderate heating is determined and increasing a speed of the evaporator fan when the requirement for increased heating is determined.

22. The method of claim 21, further comprising activating a heater to heat the supply air when the requirement for increased heating is determined.

23. The method of claim 22, further comprising operating the evaporator fan at a first speed to circulate air in the refrigerated space and adjusting the heater so that a friction heat from air circulated by the evaporator fan in the refrigerated space plus heat generated by the heater satisfies the requirement for increased heating.

24. The method of claim 21, wherein activating the evaporator fan of the refrigeration system when the requirement for moderate heating is determined includes activating the evaporator fan to circulate air in the refrigerated space at a rate where a friction heat generated by the circulated air satisfies the requirement for moderate heating.

25. The method of claim 21, wherein determining the requirement for moderate heating and the requirement for increased heating is performed at predetermined time intervals.

26. The method of claim 21, wherein determining the requirement for moderate heating and the requirement for increased heating based on a difference between the return air temperature and the supply air temperature.

- 27. The method of claim 21, further comprising:
 - determining a requirement for cooling based on the return air temperature and the supply air temperature, and
 - activating the compressor and the evaporator fan when the requirement for cooling is determined.

28. The method of claim 27, further comprising increasing a power required by the compressor when increased cooling is determined.

29. The method of claim 21, further comprising deactivating the compressor and the evaporator fan when neither heating nor refrigeration is required.

30. The method of claim 21, further comprising determining a requirement for cooling based on a set-point temperature for cargo stored in the refrigerated space.

31. A refrigeration system for use in transport using the method of claim 21.

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