A belt skiving apparatus (120) and method in which a free-standing blade-carrying carriage (122) is employed without a guiding base. The carriage has one or two pairs of rollers (124a, 126a, 124b, 126b) defining a nip(s) into which a belt end (125) to be skived is passed. The rollers press together about the belt end to securely grip the belt, with at least one of the rollers being driven through a crank arm (138). Rotation of the rollers advances the carriage relative to the belt and across the width of the belt. As the carriage is advanced, the belt (128) is passed through the rollers and through a cutting blade (134) disposed adjacent the rollers, with the blade slicing a thin strip (131) of the upper portion of the belt adjacent its upper face (130) from the remainder of the belt. An adjustable fence (200) is used to vary width of cut.
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METHOD AND APPARATUS FOR SKIVING BELT ENDS

Field of the Invention
The present invention pertains to devices for cutting a layer of material from a conveyor belt.

Background of the Invention
Conveyor belt ends are commonly joined together by mechanical fasteners which are fastened to either end of the belt and fastened to one another to connect the belt ends and form a continuous loop. In many applications, it is desirable to skive the belt to form undercuts or grooves in the upper and/or lower surfaces of the belt to receive the mechanical fasteners. Considerable problems have been encountered with currently existing methods and apparatus for skiving belt ends.

United States Patent No. 4,315,450 discloses a device which has performed satisfactorily for skiving belt ends and British patent application 2 227 703A published August 8, 1990 discloses another device proposed for skiving belts. These devices employ a guiding base to which a belt end is clamped, and along which a blade-carrying carriage is guided for reciprocal sliding movement along the base. The base represents a significant portion of the overall cost of these devices and is a large member that often must be transported to the belt location. The base can be both heavy and awkward to carry. Also, with these devices, the belt is required to be securely clamped in stationary position to the base prior to reciprocation of the carriage. The clamping of the belt to the base prior to skiving the belt is time-consuming. There is a need for a skiving apparatus and method which does not require a separate carriage-guiding base or clamping of the belt to a base, and yet produces accurate skiving of belts.
It has been a particular problem to obtain the necessary force to pull a skiving blade through tough fabric, cords, rubber, or other plastic materials in a strong tough conveyor belt. Thus, in the aforementioned United States patent there is a base secured to the belt and a winch and cable mounted on the base to exert a pull on the skiving carriage. In the aforementioned British patent, the base is provided with a long rack and the carriage is provided with a pinion gear that meshes with the rack on the base so that turning of the gear causes the carriage to travel forwardly along the base pulling the blade through the belt.

Furthermore, in the prior art devices, the blade carrying carriages experience considerable drag in their sliding movement across the belt when being pulled by a long cable or the like thereby, making manual operation more difficult and time consuming. For instance, the cable of the winch drive of United States Patent No. 4,315,450 pulls the top of the carriage in a downward direction which tends to dig the blade in deeper and to provide a downward component of force on the slidable carriage causing increased resistance to carriage travel along the base. There is a need for a belt skiving device which reduces hang-up and drag of the blade-carrying carriage and provides the force to drive the carriage blade through the belt without a large base for carrying a rack or a winch.

The upper surface of the worn belts are often uneven because of uneven wear the belt has experienced. When a base is clamped to such an uneven surface, it is more difficult to obtain a precise depth of cut and to obtain a substantial uniform depth of cut when skiving such a belt. Thus, there is a need for a device which accurately skives a belt end to produce a predetermined belt thickness at the skived portions of the belt end, regardless of unevenness of the upper surface of a belt due to uneven wear of the upper surface or the like.
Summary of the Invention

In accordance with the present invention, a belt skiving apparatus and method are provided in which a free-standing blade-carrying carriage is employed, which does not require a guiding base having a rack or a winch thereon to provide the force to cut the tough conveyor belt. This is achieved by driving the blade carrying carriage along the belt with a pair of rollers defining a nip to receive the belt and providing a driving roller mounted on the carriage that will not slip on the belt as the driving roller is turned to propel the carriage across the belt with the skiving blade cutting tough conveyor belt material. The illustrated driving roller is formed with teeth that grip the strip being skived.

Rotation of the toothed driving roller advances the carriage relative to the belt to advance the belt through the rollers and through a cutting blade disposed adjacent the rollers, whereby an upper portion of the belt passes above a horizontal portion of the blade, and a lower portion of the belt passes below the horizontal portion of the blade. As the carriage is advanced along the width of the belt by the rollers, the blade slices a narrow strip of the upper portion of the belt adjacent its upper face from the remainder of the belt. The carriage may be mounted on wheels to roll directly on a support surface, without the need for a guiding base. Since no guiding base is required, the skiving apparatus of the present invention may be significantly smaller in size and weight and also less expensive to produce than prior art skiving apparatus requiring a base.

Also, since the belt is gripped directly in the nip of the rollers of the carriage, it is not necessary to clamp the belt end in a stationary position prior to skiving the belt end. This makes skiving carried out with the skiving apparatus of the present invention significantly more simple and less time-consuming than prior art skiving apparatus.
Preferably, the carriage is provided with two sets of rollers which clamp the carriage to the belt and propel the carriage across the belt end with one set of rollers providing the driving force to cut the last portion of the belt after the belt has passed through the nip of the other set of rollers. The rollers may be driven by operation of a ratchet wrench or a power tool to shift the carriage along the belt end.

The width of the cut is adjustable by means of a guide or fence that is adjustably mounted to limit the insertion depth of the belt end into the skiving apparatus. Thus, the guide or fence can be quickly adjusted to a particular position relative to the carriage and locked by a cam and lever device at that position to provide the belt end with a cut of a predetermined width.

Still further, since the lower face of the belt bears firmly against one of the rollers during skiving, the thickness of the belt may be registered or measured from the bottom of the belt, rather than being measured from the upper surface. Hence, the skiving apparatus produces a groove of substantially uniform thickness, regardless of the presence of unevenness in the upper face of the belt prior to skiving.

In addition to overcoming the aforementioned shortcomings associated with prior art skiving apparatus, the skiving apparatus of the present invention lends itself to simple and economical manufacture, and has been found to be easy to use manually in limited space applications such as narrow mines and produce good, substantially uniform grooves.
Brief Description of the Drawings

In the drawings, wherein like elements are referenced alike:

FIG. 1 is a perspective view of a belt skiving apparatus embodying various features of the present invention;

FIG. 2 is a rear elevational view of the belt skiving apparatus of FIG. 1;

FIG. 3 is a side elevational view of the belt skiving apparatus of FIG. 1;

FIG. 4 is a side elevational view of the pressure roller of the apparatus of FIG. 1;

FIG. 5 is an enlarged, partial top view of the pressure roller showing its pivotal connection to the carriage;

FIG. 6 is an enlarged, partial front elevational view showing the engagement of the driving roller with the belt;

FIG. 7 is an enlarged, perspective view of the cutting blade of the apparatus of FIG. 1;

FIG. 8 is partial elevational view of the front side of the vertical wall of the carriage, showing the hourglass-shaped recess in the wall and the aperture through the wall;

FIG. 9 is a perspective view of preferred embodiment belt skiving apparatus embodying various features of the present invention;

FIG. 10 is a front elevational view of the skiving apparatus of FIG. 9, shown with the pinch rollers in their release position;

FIG. 11 is a front elevational view of the skiving apparatus of FIG. 9, shown with the pinch rollers in their clamping position;

FIG. 12 is a rear elevational view of the skiving apparatus of FIG. 9, shown with the pinch rollers in their clamping position;
FIG. 13 is a top elevational view of the skiving apparatus of FIG. 9;

FIG. 14 is an enlarged top elevational view of the pivotal bracket in the skiving apparatus of FIG. 9;

FIG. 15 is an enlarged perspective view of the blade in the skiving apparatus of FIG. 9;

FIG. 16 is perspective view of the skiving apparatus of FIG. 9, shown with the pivotal brackets removed;

FIG. 17 is a perspective view of the skiving apparatus of FIG. 9, shown with the fence moved forward relative to the fence position of FIG. 9;

FIG. 18 is a rear elevational view of the skiving apparatus of FIG. 9, showing a manually driven ratchet arm connected to a drive roller;

FIG. 19 is a rear elevational view of the skiving apparatus of FIG. 9, showing a pneumatically powered ratchet arm connected to the roller; and

FIG. 20 is a top plan view of the body of a slightly modified version of the skiving apparatus of Fig. 9, including a fence cam-locking mechanism;

FIG. 21 is a front elevational view of the skiving apparatus of Fig. 20;

FIG. 22 is a rear elevational view of the skiving apparatus of Fig. 20; and

FIG. 23 is an enlarged elevational view showing the handle of the skiving apparatus of FIGS. 20-22.

Detailed Description of the Preferred Embodiments

As shown in the drawings for purposes of illustration, the invention is embodied in a skiving apparatus 20 that has a carriage 22 carrying a skiving blade 34 for cutting a strip 40 from the belt 28. The carriage may have wheels for rolling across a supporting surface, such as a workbench or conveyor support plate (not shown) and the belt is passed through a nip formed between a pair of rollers 24 and 26. In accordance with
the present invention, the force for driving the cutting blade 34 through the belt is from the pair of rollers 24 and 26 at least one of which is provided with a non-slip surface, such as tooth surfaced, and is driven. As shown in FIG. 6, the first illustrated driving roller 24 has teeth 52 that puncture an upper surface area of the belt, which area is being skived. A lower cost, non-slip surface for the rollers is shown in FIGS. 9-11 where the rollers have an outer corrugated surface formed with parallel, pointed outer elongated teeth 184 between depressions. The rollers shown in FIGS. 9-11 are extruded aluminum rods that are severed into the rollers. The driving roller is located on the carriage closely adjacent the cutting blade and a pinch roller 26 for the driving roller, thereby eliminating the long winch cables or racks on long bases of the prior art skiving devices.

More specifically the driving roller 24 and pressure roller 26 are pressed together against respective upper and lower faces 30 and 32 of the end portion 33 of the belt 28 to clamp the belt therebetween. The skiving blade 34 has a generally horizontal blade portion 36 adjustable to any selective position between the rollers 24 and 26. The driving roller 24 is manually rotatable through a crank arm 38. Rotation of the driving roller 24 advances the carriage 22 relative to the belt 28 to advance the belt 28 through the rollers 24 and 26 and through the cutting blade 34, with an upper strip or portion 40 of the belt 28 passing above the horizontal portion 36 of the blade 34 and a lower portion 42 of the belt 28 passing below the horizontal portion 36 of the blade 34. The blade 34 slices the upper portion 40 of the belt 28, adjacent its upper face 30, from the remainder of the belt 28. As the driving roller 24 is rotated, the carriage 22 is advanced across the width of the belt 28 to advance the blade 34 relative to the belt 28 to skive the belt across its width.
The rollers 24 and 26 engage the belt directly so that no separate base is required as in the prior apparatus. Since the belt is clamped securely by the toothed roller 24 and pinch roller 26 which are disposed adjacent the blade 34, there is little slippage or play of the belt 28 with respect to the blade as the belt is being skived, which allows skiving of thinner upper belt portions than previously attainable while also producing a good, uniform groove in the belt.

Also, the carriage 22 is free-standing, rather than moving within a channel of a base as in the prior art; so that the potential problem of hang-up or sticking of the carriage within the channel is eliminated with the structure of the present invention. Still further, the lower face 32 of the belt 28 bears against the pressure roller 26 during cutting of the belt 28, and the blade 34 is adjustably positionable to a selective distance from the pressure roller 26 to cut the belt to a selective thickness as measured from the lower face 32 of the belt, to overcome variations in thickness associated with wear of the upper face of the belt. Hence, the skiving apparatus of the present invention can cut off an uneven upper face 30 of a belt 28 and produce a groove with the belt having uniform thickness at the groove despite the unevenness of the upper face 30 of the belt 28 at the groove prior to skiving.

The illustrated carriage 22 is a generally L-shaped plate which defines integral vertical wall 44 and horizontal wall 46. The carriage 22 may be formed of metal plates, but for reduced manufacturing cost may alternatively be produced of plastic. The driving roller 24 is mounted at a fixed position on the front side 50 of the vertical wall 44. An L-shaped bracket 48 extends from the front side 50 of the vertical wall 44 to rotatably support the driving roller 24 about a fixed rotational axis. As best seen in FIGS. 1 and 3, the driving roller 24 is preferably formed in one piece with
stub axles 52a and 52b (FIG. 6) on opposite ends of a larger diameter center portion 52c that has the integral teeth 52 thereon. The stub axles 52a and 52b are each received in a respective bearings 54 of the vertical wall 44 and a bearing 56 of the L-shaped bracket leg 58 which mounts the roller for rotation about its fixed rotational axis. The axle 52a received in the bearing 54 of the vertical wall 44 is an elongated axle which extends completely through the vertical wall and is received in the base 53 of the crank arm 38 and secured by screw 55 for driving rotation of the driving roller 24 as described further below. Alternatively, the elongated axle 52a mounts a hex nut which can be received in a socket as in a conventional manually operated ratchet arm or pneumatically powered ratchet arm, as will be more fully described herein.

To accommodate belts of various thicknesses, the pressure roller 26 is disposed on a pivotal bracket 60 to allow selective adjustment of the height of the pressure roller 26 relative to the driving roller 24. By pivoting the bracket 60, the space between the pressure roller 26 and the driving roller 24 may be varied as desired for a given belt thickness, as explained further below.

As best seen in FIGS. 1 and 8, the front side 50 of the vertical wall 44 of the carriage 22 has a generally hourglass-shaped recess 62 formed therein. The pressure roller supporting bracket 60 pivots about a pivot pin 65 which extends into a bore 68 in the bracket 60 and a bore 66 in the vertical wall 44. The portion of the bracket 60 disposed on one side of the pivot pin 65 is an adjusting arm portion 72, and the portion of the bracket 60 disposed on the other side of the pivot pin 65 is a pressure roller supporting portion 74 having a U-shape with the pressure roller 26 supported on either end by legs 71 of the U-shaped portion 74.

The inner side 64 of the respective portions 72 and 74 of the pivotal bracket 60 are disposed in
respective portions 76 and 78 of the recess 62 for oscillatory pivotal movement of the inner sides of the bracket portions 72 and 74 within respective recess portions 76 and 78. In the illustrated embodiment, the pressure roller 26 is disposed generally beneath the driving roller 24, and pivotal movement of the bracket 60 varies the distance between the driving roller 24 and the pressure roller 26 to accommodate belts of different thickness. To adjust the position of the pressure roller 26 with respect to the driving roller 24, the adjusting arm portion 72 of the bracket 60 has a short leg 80 which extends generally perpendicularly with respect to the inner side 64 of the bracket 60, and extends into an aperture 82 in the vertical wall 44 (see FIGS. 1 and 5). A threaded pivotal bracket adjusting member 84 is threadably received within a threaded bore 86, which extends generally vertically and parallel to the front side 50 of the vertical wall 44. The upper end 89 of the threaded pressure roller adjusting member 84 extends upwardly from the upper end 90 of the vertical wall 44, and the lower end 94 of the threaded member 84 extends down into the aperture 82 in the vertical wall 44 and bears against the upper side 92 of the leg 80 of the adjusting arm 72.

Accordingly, adjustment of the nip between the pressure roller 26 and the driving roller 24 and the pressure needed for the teeth 52 to puncture the belt is obtained for a given belt thickness by simple manual rotation of the handle 88 provided at the upper end 89 of the threaded adjusting member 84. Rotation of the handle 88 rotates the threaded adjusting member 84, which moves the lower end of the threaded member 84 vertically. Hence, clockwise rotation of the handle 88 moves the lower end 94 of the threaded member 84 downwardly to force the adjusting arm portion 72 of the bracket 60 downwardly, which in turn pivots the bracket 60 about pivot pin 65 to raise the pressure roller 26 upwardly and
narrow the space between the pressure roller 26 and the driving roller 24. Similarly, counter-clockwise rotation of the handle 88 moves the lower end 94 of the threaded member 84 upwardly to allow the adjusting arm portion 72 of the bracket 60 to move upwardly, which in turn pivots the bracket 60 about pivot pin 65 to lower the pressure roller 26 downwardly, and enlarge the space between the pressure roller 26 and the driving roller 24. FIG. 2 illustrates the variable positioning of the pressure roller.

In practice, the handle 88 may be rotated counter-clockwise so that the space between the rollers 24 and 26 is greater than the thickness of the belt 28 to be skived. A leading end of the belt 28 is inserted between the rollers 24 and 26 and held in place as the handle 88 is rotated clockwise to raise the pressure roller 26 upwardly into abutment with the lower face 32 of the belt 28. The handle 88 is then rotated clockwise further, to securely clamp the belt 28 between the rollers 24 and 26 and to cause the teeth 52 to puncture the belt strip to be severed. With the belt 28 firmly engaged between the rollers 24 and 26, rotation of the driving roller 24 advances the belt 28 through the rollers 24 and 26, as described further below.

The cutting blade 34 is preferably disposed directly adjacent the rollers 24 and 26 to slice off a layer of the belt 28 adjacent its upper face 30, as the rollers 24 and 26 grip and advance the blade 34 across the width of the belt 28. Unlike the aforementioned prior art skiving apparatus in which the belt is either clamped to a base at its edges prior to skiving or nailed to a base across its entire width prior to skiving, the belt skiving apparatus of the present invention does not require such time-consuming clamping preparations prior to carrying out skiving of the belt, as the same rollers 24 and 26 which drive the carriage 22 across the belt 28 also clamp the belt securely during cutting. That is,
the rollers 24 and 26 clamp the belt securely directly adjacent the blade 34 during skiving. The rollers 24 and 26 maintain clamping engagement of the belt 28 directly adjacent the blade 34 throughout advancement of the carriage 22 across the width of the belt 28 so that the belt is gripped securely by the rollers 24 and 26 directly adjacent the blade 34 throughout skiving of the belt by the blade 34. This has been found to provide significantly improved accuracy in the grooves formed by skiving, which allows thinner upper layers to be accurately removed than previously obtainable.

The illustrated blade 34 is U-shaped having a central, generally horizontal blade portion 36 and two spaced vertical blade portions 102 extending upwardly at and from opposite ends of the central horizontal blade portion 36. Other shapes of blades, such as a horizontal blade, may be used instead of the U-shaped blade. The forward edges of the horizontal and vertical blade portions 36 and 102 are sharp cutting edges. The vertical blade portions 102 fit in recesses in opposite faces of a blade holding block 103 and are secured to the blade holding block 103 by spaced bolts being threaded into the block through openings in the blade. The block 103 is supported in turn by bolts threaded into taps in the block and protruding through slots 105 in the vertical wall 44 of the carriage 22. Upon tightening the bolts against the slides, the block can be held with the horizontal blade portion 36 at any adjusted depth below the upper face 30 of the belt so as to cut away the upper portion 40 of the belt 28 and form the desired groove adjacent the belt end. To assist in holding the block while the bolts are being tightened, a holding screw 107 may be employed with its lower end threaded into the top of the holding block 103. The block may be held in place with one hand by wing nut 109 while the operator tightens the bolts to secure the blade with the other hand.
As discussed above, the driving roller 24 is operatively coupled to the base 53 of the crank arm 38 through axle 52a. Hence, the driving roller 24 may be rotated by the operator manually gripping the handle 88 of the crank arm 38 and rotating the crank arm 38. Manifestly, rather than a direct connection between the driving roller 24 and the crank arm 38, the crank arm 38 may be operatively connected to the driving roller 24 through gears to obtain a mechanical advantage. However, it has been found that the force required to advance the belt 28 through the blade 34 to skive the belt, with the structure of the belt skiving apparatus of the present invention, is sufficiently small that no gears are required. Also, a motor may be employed to drive the drive roller, however, due to the extra space which external drives require and the added production cost, external drives are not desired. Hence, to minimize production costs, no gears or external, automated roller driving equipment are employed in the illustrated embodiment of the invention. Nevertheless, as previously mentioned, it is possible to use a manually operated ratchet arm or a pneumatically powered ratchet arm in place of the crank arm 38 to drive the drive roller without the additional space and expense as required with external drives as described above.

The driving roller 24 preferably has a non-slip surface, preferably such as short spikes or teeth 52 extending about its periphery. As best illustrated in FIG. 6, the spikes 52 dig into the upper face 30 of the belt 28 when the belt 28 is clamped between the rollers 24 and 26 to securely grip the belt and prevent slippage of the belt with respect to the driving roller 24. Hence, upon rotation of the crank arm 38, the drive roller 24 rotates and its teeth 52 dig into and grip the upper face 30 of the belt as the drive roller rotates, to securely grip the belt and advance the belt with respect to the blade 34. Other alternatives to teeth may be
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roughened surfaces or the like to prevent slippage of the
drive roller 24 on the belt surface as the drive roller
is turned.

It is important, in order to produce a uniform
groove in the belt with the skiving apparatus 20 of the
present invention, that the end of the belt being skived
be maintained flush against the front side 50 of the
vertical wall 44 throughout advancement of the carriage
22 along the width of the belt, and not migrate away from
the front side 50 of the vertical wall 44 as the carriage
is driven across the width of the belt. This may be
carried out in any of several ways. For instance,
additional rollers may be provided to guide and hold the
belt directly adjacent the front side 50. Alternatively,
the rollers 24 and 26 may be tapered in the direction of
the vertical wall 44 to urge the belt 28 in the direction
of the vertical wall 44.

Also, to allow for variability of the width of
the groove cut into the belt means may be provided for
adjusting the position of the front side 50 of the
vertical wall 44 and/or the spacing of the blade 34 from
the vertical wall. For instance, an adjustable thin
plate may be provided adjacent and generally parallel to
the vertical wall 44. The plate may be moved toward and
away from the vertical wall 44 to allow selective
variability of the width of the groove from the end 33 of
the belt 28. The position of the blade 34 may also be
adjustable for variation of the width of the groove
formed in the belt, with the blade being movable closer
to the front side 50 of the vertical wall 44 for
formation of thinner grooves and movable further from the
front side 50 of the vertical wall 44 for formation of
wide grooves.

In carrying out skiving of a conveyor belt in
accordance with the present invention, an end portion of
a belt is passed between a pair of rollers which clamp
about the belt. One of the rollers is a driving roller.
which is rotated to advance the blade-carrying carriage with respect to the belt across its width. The illustrated carriage has rollers which roll across the width of the belt as the belt is advanced through the rollers. The illustrated carriage has wheels, but the carriage may be constructed without any wheels. A cutting blade is disposed adjacent the rollers, and positioned such that the rollers force an upper portion of the belt above the blade and a lower portion of the belt below the blade. The forward edge of the blade thus slices off the upper part of the belt from the remainder of the belt as the carriage is advanced along the width of the belt.

While the skiving apparatus 20 described above has generally been found to be well-suited for skiving a belt, it was found that the skiving apparatus 20 did not always drive the blade past the final, trailing end portion of the belt, leaving a small end portion of approximately 3/16 inch of the belt uncut. That is, with the skiving apparatus 20 near the completion of skiving a belt across its width from a leading side of the belt to a trailing side of the belt, the trailing side of the belt passes through, and out of engagement with the driving roller 24 prior to reaching the cutting blade 34. Therefore, there is no driving of the final trailing side portion of the belt past the blade, and the short portion at the trailing side of the belt remains uncut, requiring a separate operation to manually cut the final trailing end portion of the belt. There is a need for a belt skiving apparatus which cuts a strip from a belt along the entire width of the belt adjacent at its end, from its leading side completely to its trailing side.

Also, with the belt driven by the toothed roller 24 having generally uniformly distributed teeth about its periphery as discussed above, the belt 28 may migrate away from the front side 50 of the vertical wall 44 and may become angled with respect the front side 50 of the
vertical wall 44. This may result in unevenness in the width of the strip cut from the belt. Additionally, the production of the toothed roller 24 is relatively expensive. Accordingly, there is a need for an arrangement for driving the belt which prevents the belt from becoming askew as it is advanced with respect to the cutting blade to assure accurate skiving of the belt. It is also desirable to provide such an arrangement in which the driving roller may be produced less expensively than the toothed roller discussed above.

A preferred embodiment of a skiving apparatus which satisfies the above objectives is illustrated in FIGS. 9-13, and referred to generally by reference numeral 120. As with the skiving apparatus 20 discussed above, the preferred skiving apparatus 120 has a free-standing carriage 122. The carriage 122 is a generally L-shaped member which defines an integral vertical wall 144 and horizontal wall 146. The carriage may have wheels 147 for rolling across a supporting surface such as a workbench or conveyor support plate.

The carriage 122 of the preferred embodiment carries a J-shaped skiving blade 134 for cutting a strip from the end 125 of a belt 128, as discussed further below. The portion of the belt 128 adjacent its end 125 is passed through a pair of nips defined by respective pairs of rear and forward driving and pressure rollers 124a and 126a, and 124b and 126b, with a forward nip defined by the forward driving and pressure rollers 124b and 126b disposed closely ahead of the blade 134 and a rear nip defined by the rear driving and pressure rollers 124a and 126a disposed rearward of the blade 134. The rear and forward driving rollers 124a and 124b and respective rear and forward pressure rollers 126a and 126b, which together define the aforementioned forward and rear nips, are pressed against respective upper and lower faces 130 and 132 of the end portion 133 of the belt 128 which is to be skived to securely clamp the belt
between the respective driving and pressure rollers at both the forward and rear nips. Hence, with the belt 128 securely clamped between both the forward and rear pairs of driving and pressure rollers, the carriage may be driven relative to the belt at two separate locations, one forward and one rearward of the blade 134, upon rotation of the drive rollers 124a and 124b. Therefore, during a skiving operation, after an upper strip 131 has been sliced from the belt 128 from its leading end 127 to near its trailing end 129, and the trailing end 129 of the belt passes between the forward driving roller 124b and forward pressure roller 126b and out of engagement therewith, the belt continues to be pulled forward by the rear driving roller 124a and rear pressure roller 126a to pull the trailing end 129 of the belt 128 past the blade 134 to cut the final part of the strip 131 at the trailing side 129 of the belt 128.

To drive the rear and forward driving rollers 124a and 124b together, the driving rollers 124a and 124b are preferably interconnected by a chain 121 (FIG. 12). Herein, the rear and forward driving rollers 124a and 124b rotate together by manually rotating a crank arm 138 attached to one of the drive rollers 124, preferably the rear drive roller 124a. Rotation of the crank arm 138, or operation of the previously mentioned ratchet arm to be more fully described herein, causes both the forward and rear pair of rollers to advance the carriage 122 and the cutting blade 134 mounted to the carriage 122 relative to the belt 128, with an upper strip or portion 131 of the belt 128 passing above the blade 134 and a lower portion 142 of the belt 128 passing below the blade 134, to slice away an upper strip 131 of the belt 28 from the forward side 127 of the belt 128 to the trailing side 129 of the belt. After the trailing side 129 of the belt 128 has been passed through the forward driving roller 124a and forward pressure roller 126a, continued rotation of the crank arm 138 effects continued rotation of the
forward drive roller 124a for continued driving of the carriage relative to the belt 128 to drive the blade 134 past the rear side 129 of the belt. The interconnection of the drive rollers 124a and 124b eliminates the necessity of removing the crank arm 138 from one of the drive rollers and reattaching it to the other drive roller to carry out the final cut of the strip 131 from the belt.

The pair of nips forward and rearward of the blade 134 overcome several shortcomings associated with a single drive roller and pressure roller. As discussed above, a single drive roller and pressure roller rearward of the blade 134 does not drive the final trailing side of the belt past the blade. A single drive roller and pressure roller forward of the blade 134, while being suitable for pulling the entire trailing side of the belt past the blade, would make it difficult to initiate the skive at the forward side 127 of the belt. Additionally, as discussed above, the provision of a nip closely adjacent the blade 134 provides accuracy in skiving operations which is not attainable with nips spaced further from the blade.

The combination of a first nip defined by a forward driving roller 124b and pressure roller 126b closely adjacent and forward of the blade 124 and a second nip defined by a rear driving roller 124a and rear pressure roller 126a provides accurate skiving over the entire width of the belt 128, from its leading side 127 to its trailing side 129. Specifically, the forward nip defined by the forward driving roller 124b and forward pressure roller 126b engages the forward side 127 of the belt 128 initially and advances the belt past the blade 134 from the forward side 127 of the belt 128 toward its trailing side 129. The forward driving roller 124b and pressure roller 126b advance the forward side 127 of the belt 128 into the rear nip defined by the rear driving roller 124a and rear pressure roller 126a, whereupon both
the forward and rear pairs of rollers drive the belt together. The rear and forward driving rollers 124a and 124b continue to drive the belt together until the trailing side 129 of the belt 128 passes through the forward driving roller 124b and pressure roller 126b, and out of engagement therewith. Thereafter, the rear driving roller 124a continues to drive the belt 128 to pull the small uncut portion of the strip 131 at the trailing side of the belt past the blade 124 to effect the final cutting of the strip 131 from the belt 128.

Since the forward nip defined by the forward drive roller 124b and forward pressure roller 126b is situated in close proximity to the blade 134, the end portion of the strip 131 remaining uncut at the trailing side 129 of the belt after the trailing side of the belt passes between the forward drive roller 124b and forward pressure roller 126b is small.

In the illustrated embodiment, the rear and forward driving rollers 124a and 124b are supported by a supporting bracket 150 (FIGS. 9 and 16) which extends generally horizontally forward of the vertical wall 144. In the illustrated embodiment, the vertical wall 144 is provided with respective channels 146 in its rear and forward ends 148 and 149 for receiving respective legs 152 of the supporting bracket 150. The legs 152 are secured in their respective channels 146 to support the roller engaging bar portion 154 of the supporting bracket 150 horizontally and in spaced relation from the vertical wall 144, as illustrated in FIG. 9.

The drive rollers 124a and 124b have respective stub axles 156a and 156b. The stub axles 156a and 156b of the drive rollers 124a and 124b are each received in respective bearings (not shown) of the vertical wall 144 and bearings 158 of the roller engaging bar portion 154 of the supporting bracket 150 which mounts the drive rollers 124a and 124b for rotation about their respective fixed rotational axes. The stub axle 156a of the rear
drive roller 124a received in the bearings of the vertical wall 144 is an elongated axle which extends through the vertical wall 144. The portion of the elongated axle 156a of the rear drive roller 124a which extends through the vertical wall 144 of the carriage is receivable in the base 140 of a crank arm 138 for driving rotation of the rear drive roller 124a. The axles 156a and 156b of the rear and forward drive rollers 124a and 124b have respective sprocket wheels 162a and 162b, preferably of substantially equal diameter to one another, which are interconnected by the chain 121 to drive both the rear and forward drive rollers 124a and 124b simultaneously. As previously stated, manual rotation of the crank arm 138 rotates both of the drive rollers 124a and 124b simultaneously.

Alternatively, the portion of the elongated axle 156a of the rear drive roller 124a which extends through the vertical wall 144 of the carriage mounts a hex nut 230, as seen in FIGS. 18 and 19. By mounting the hex nut 230 to the portion of the elongated axle extending through the vertical wall, a conventional ratchet arm 232 and socket 234 can be employed to drive the drive roller 124a. Referring to FIG. 18, the ratchet arm 232 can be cranked and ratcheted to rotate the socket 234 and hex nut 230 received therein to thereby rotatably drive the drive roller 124a. Similarly, as seen in FIG. 19, a pneumatic or other power actuated tool 236 can be utilized. The tool 236 is connected at one end 238 to an air supply line 240 and at the other end 242 to a socket 244 adapted to receive the hex nut 230 therein. Thus, operation of the pneumatically driven ratchet tool 236 rotates the socket 244 and hex nut 230 to drive the rollers and propel the carriage along the belt ends.

With reference to FIG. 13, in the illustrated embodiment, the vertical wall 144 has a centralized recess 159 adjacent its rear face 157. The sprocket wheels 162 and chain 121 are disposed in the recess 159
of the vertical wall 144, so that the vertical wall 144 shields the sprocket wheels 162 and chain 121.

As with the skiving apparatus 20 discussed above, the gap between the pressure rollers 126a and 126b and their respective driving rollers 124a and 124b is variable to accommodate belts 128 of various thicknesses. To this end, the pressure rollers 126a and 126b are disposed on respective pivotal brackets 160 (see FIG. 14), which are, in turn, mounted on respective pivot pins 161 extending horizontally forward from the vertical wall 144, to allow selective adjustment of the height of the pressure rollers 126a and 126b relative to their respective driving rollers 124a and 124b.

As best seen in FIGS. 11 and 16, the vertical wall 144 has a recess 168 in its front face 151 over a central, lower portion thereof in which the legs 170 of the pivotal brackets 160 are disposed, and within which the legs 170 of the pivotal brackets 160 move as the brackets 160 are pivoted about their respective pivot pins 161.

To assure that substantially equal belt clamping force is exerted on the belt by the forward and rear pairs of rollers 124a, 126a and 124b, 126b the pivotal brackets 160a and 160b are both actuated by a common, centrally disposed actuating member 164 as illustrated in FIGS. 10-12.

The actuating member 164 extends vertically through the vertical wall 144 of the carriage 122 and has a threaded shaft 166 which is threaded in a threaded bore in the vertical wall 144 and moves vertically upon manual rotation of the integral turning knob 173 of the actuating member 164. The lower end 174 of the actuating member 164 extends into the recess 168 and has a camming head 176 which is preferably of low friction plastic having a rounded bearing surface 180 which bears simultaneously against the legs 170 of both of the pivotal brackets 160 which extend into the recess 168.
Upon manual clockwise rotation of the turning knob 173, the camming head 176 is moved downwardly to simultaneously force both the legs 170 of the pivotal brackets 160 downwardly within the recess 168. The pivotal brackets 160 are simultaneously pivoted about their respective pivot pins 161 to move the pressure rollers 126a and 126b from their release position illustrated in FIG. 11 to their clamping position illustrated in FIG. 10 to clamp the belt between the pressure rollers 126a and 126b and their respective driving rollers 124a and 124b. The leading end 127 of the belt 128 may be inserted between the forward and rear, drive and pressure rollers while they are in their release position of FIG. 11. The actuating member 164 may then be screwed down until the belt 128 is securely clamped between the forward drive roller 124b and forward pressure roller 126b and the rear drive roller 124a and pressure roller 126a. The more the actuating member 164 is screwed downwardly into the vertical wall 144, the greater the clamping force exerted by the rollers 124a, 126a and 124b, 126b on the belt, with the clamping force being exerted on the belt by the front and rear pairs of rollers remaining substantially equal. The horizontal wall 146 of the carriage 122 has an aperture 188 (FIG. 16) therein beneath the recess 168, for allowing the legs 170 of the pivotal brackets 160 to move downwardly past the upper surface 186 of the horizontal wall 146. The skiving blade 134 which cuts the strip 131 from the belt 128 is preferably J-shaped, having a horizontal portion 188 adjacent the vertical wall 144 and a vertical portion 190 spaced from the vertical wall 144, which portions 188 and 190 are interconnected by an integral intermediate arcuate portion 192. The forward edge 194 of the blade 134 is preferably sharp and continuous to continually slice the strip 131 from the belt 128 as the belt is advanced past the forward edge 194 of the blade 134. The
blade 134 is secured in a selective, stationary position prior to and during skiving. (See FIG. 15.)

The height of the horizontal portion 188 of the blade 134 above the horizontal wall 146 determines the thickness of the belt strip 131 being cut and thereby, the depth of the groove formed at the belt end. The height of the blade 134, and hence the height of the horizontal portion of the blade 134, is selectively variable by rotation of the knob 195 (FIG. 10) of a blade adjusting means 196.

As best seen in FIGS. 9 and 10, the blade adjusting means 196 comprises a threaded shaft 196a threaded in a bore 196b of a stationary frame plate 189 which is spaced by a space 191 from the top of a blade carrying block 192 which has its vertical slides guided for vertical sliding movement in guideways 193 in the stationary frame. The lower end of the shaft 196a is captured in the blade-carrying block 192 vertically as the shaft is turned to thereby move the blade up or down.

The blade adjusting member 196 is threadably engaged with the vertical wall 144 of the carriage 122 for selective upward and downward movement of the blade adjusting member 196 and the blade 134 engaged thereto upon rotation of the knob 195.

The distance of the vertical portion 190 of the blade 134 from the belt end 125 determines the width of the strip 131 cut from the belt 128, i.e. the width of groove formed in the belt, adjacent the belt end 125. In the preferred and illustrated embodiment, rather than adjust the horizontal position of the blade, the position of the belt end 125 is variable to selectively vary the width of the groove formed in the belt.

In the preferred embodiment of the invention, to allow for simple and accurate variations of the width of the strip cut from the belt end, a fence 200 (FIGS. 13 and 17) is provided which comprises a flat, vertical plate extending substantially parallel to the front face
151 of the vertical wall 144 of the carriage 122. The position of the fence 200 is adjustable for variation of its distance from the front face 151 of the vertical wall 144; and hence, variation of the distance of the vertical portion 190 of the blade 134 from the end 125 of the belt 128 being skived, and hence the wider the groove formed in the belt adjacent its end 125. As best seen in from a comparison of FIGS. 9 and 17, the fence 200 is spaced from the vertical wall 144 in FIG. 17 and the fence 200 is abutted against the vertical wall 144 in FIG. 9. Thus, it will be seen that the further the fence 200 is positioned from the front face 151 of the vertical wall 144 during skiving, the narrower the strip 131 cut from the belt; and, the closer the fence 200 is positioned to the front face 151 during skiving, the wider the strip 131 cut from the belt, and hence the narrower the groove formed in the belt adjacent its end 125.

To allow for selective positioning of the fence 200 for a desired width of groove cut, the fence 200 is slidably supported between the spaced legs 152 of the supporting bracket 150 for sliding adjustment toward and away from the front face 151 of the vertical wall 140. As best seen in FIG. 9, the fence 200 has two vertical surfaces 201 and a horizontal surface 203 for sliding engagement with the upper portion of the forward leg 152 of the bracket 150. These surfaces 201 and 203 define a rectangular aperture 202 near its forward end 204 for receiving the rear leg 152 of the supporting bracket 150 therethrough. The fence 200 has a horizontal shoulder 206 at its rear end 208 in sliding engagement with the upper surface 210 of the rear leg 152 of the supporting bracket 150. Hence, the fence 200 is slidable along the pair of legs 152 of the supporting bracket 150 toward and away from the front face 151 of the rear wall 144. The fence 200 may be secured in a selective stationary position along the legs 152 of the support bracket 150 by providing a plurality of horizontally spaced holes 211
(FIG. 16) in the forward surface 212 of the forward leg 152, and providing a bracket mounted to the front face 210 of the fence 200 with the bracket having a hole at a vertical height corresponding to the vertical height of the holes in the leg 152. After the fence 200 has been slid to its desired position along the legs 152, the hole in the bracket mounted to the fence will be in registration with one of the horizontally spaced holes 211 of the rear leg 152, and a pin 213 may then be inserted through the hole of the bracket and into the corresponding hole of the leg 152 to secure the fence 200 in a stationary position along the legs 152. Manifestly, a wide variety of other engaging means may be employed for securing the fence 200 in a selective, stationary position along the legs 152.

In the illustrated embodiment, the fence 200 has forward and rear arcuate cutout portions 212 and 214 (FIG. 17) for accommodating the respective forward and rear drive rollers 124a and 124b, and a central cutout portion 216 for accommodating the blade 134. Hence, the cutout portions 212, 214 and 216 allow free forward and rearward sliding movement of the fence 200 along the legs 152 without interference of the fence 200 with the drive rollers 124a and 124b and the blade 134. The fence 200 extends from a rear end 208, which is generally coterminous with the rear end 148 of the vertical wall 144, to a forward end 204 which extends beyond the forward end 149 of the vertical wall 144, so that the fence 200 extends significantly forward of the forward driving roller 124b. The portion of the fence 200 which extends forward of the forward leg 152 extends downwardly to a vertical height below the drive rollers 124 to define a forward guiding surface portion 222 of the forward face 220 of the fence 200.

The provision of the forward guiding surface portion 222 significantly forward of the forward driving roller 124b serves to provide an elongated guiding
surface for guiding of the end 125 of the belt 128 prior to the belt 128 being passed into the forward nip between the forward pair of driving and pressure rollers 124b and 126b to provide accuracy in alignment of the belt 128 as it enters the forward nip between the forward pair of rollers. The fence 200 also has a rear guiding surface portion 224 which also extends downwardly to a vertical height below the drive rollers 124. During skiving, the end 125 of the belt 128 is maintained flush against both the forward and rear guiding surface portions 222 and 224 of the front face 220 of the fence 200 to maintain the end 125 of the belt substantially perpendicular to the axes of the drive rollers 124a and 124b as the belt is guided beneath the drive rollers 124a and 124b.

It is preferred to provide a means to maintain the end 125 of the belt 128 being skived flush against the front face 220 of the fence 200 throughout advancement of the carriage 122 along the width of the belt 128, and prevent migration of the belt end 125 away from the front face 220 of the fence 200 as the carriage is driven across the width of the belt. To this end, the illustrated means is in the form of spiral threads 226 (FIG. 14) on the peripheral surface of the pressure rollers 126a and 126b. The spiral threads 226 of the pressure rollers 126a and 126b draw the belt 128 in the direction of the front face 220 of the fence 200 as the belt 128 is driven between the front pair of rollers 124a and 126a and the rear pair of rollers 124b and 126b, with the pressure rollers 126a and 126b in their clamping position. By way of illustrative example, a left-handed spiral thread having a pitch of approximately 3.5 inch was found to provide good results in drawing the belt in the direction of the fence 200, and maintaining the end 125 of the belt flush against both the central and rear guiding surface portions 222 and 224 of the front face 220 of the fence 200 as the carriage 122 is advanced.
along the width of the belt. This provides improved accuracy in the skiving of belt ends.

The driving rollers 124a and 124b are preferably fluted about their periphery to define a plurality of longitudinally extending parallel flutes 182 and longitudinally extending elongated teeth 184 about their periphery. The elongated teeth 184 firmly grip the belt 128 for accurate driving of the carriage 122 relative to the belt without slippage of the belt between the drive rollers and pressure rollers. The drive rollers 124 may be made of aluminum or plastic and the straight, parallel flutes and teeth of the driving rollers 124a and 124b allow the drive rollers to be produced by extrusion at low cost.

To carry out skiving of a belt with the skiving apparatus 120, initially the end 125 of the belt 128 to be skived is placed flush against the rear guiding surface portion 222 of the front face 220 of the fence 200 to properly align the belt end 125. With the belt end 125 maintained flush against the rear guiding surface portion 222, the carriage 122 is rolled on its wheels 147 toward the belt to bring the leading side 127 of the belt 128 into the nip between the rear driving roller 124b and the rear pressure roller 126b. The actuating member 166 is rotated clockwise to raise the rear pressure roller 126b to firmly clamp the belt 128 between the rear drive roller 124b and the rear pressure roller 126b. As discussed above, in the preferred embodiment, the adjustment of the height of the rear pressure roller 126b effects the simultaneous adjustment of the height of the forward pressure roller 126a. With the leading end 127 of the belt 128 engaged by the rear drive roller 124b and pressure roller 126b, the crank arm 138 may be rotated to simultaneously rotate the front and rear drive rollers 124a and 124b. The rotation of the rear drive roller 124b advances the carriage 122 toward the trailing side 129 of the belt 128, with the blade 134 slicing a strip
131 from the belt 128 adjacent its end 125 as the carriage 121 to which the blade mounted is advanced relative to the belt.

The forward pair of rollers 124a and 124b advance the carriage 122 relative to the belt 128, to move the leading end 127 of the belt 128 past the cutting blade 134 to the rear pair of driving and pressure rollers 124a and 126a. Thereat, the leading end 127 of the belt 128 passes into the nip between the rear pair of rollers 124a and 126a, at which point the belt is driven by both the forward pair of rollers 124a and 126a and the rear pairs of rollers 124b and 126b. The end 125 of the belt 128 is maintained flush against the front face 220 of the fence 200 throughout the skiving operation. Both the forward and rear pairs of rollers advance the carriage with respect to the belt to slide off a strip from the belt adjacent its upper surface, until the trailing end of the belt passes through the forward pair of rollers. After the trailing side 129 of the belt 128 passes out of engagement with the forward pair of rollers 124b and 126b, the belt still remains firmly gripped by the rear pair of rollers 124a and 126a which continue to advance the carriage 122 and the blade 134 mounted thereto relative to the belt 128, to advance the trailing side 129 of the belt past the blade 134 to cut the strip 131 completely to the trailing side 129 of the belt 128. That is, after the trailing side 129 of the belt 128 passes out of engagement with the forward drive roller 124b and pressure roller 126b, there remains a small uncult portion of the strip 131 adjacent the trailing side 129 of the belt due to the gap between the forward or leading edge 194 of the blade 134 and the forward drive roller 124b. That is, near the end of a skiving operation, the trailing side 129 of the belt 128 passes through the forward pair of rollers 124b and 126b, and out of engagement with the forward driving roller 124b,
whereafter the forward driving roller 124b no longer advances the belt relative to the carriage 122.

Hence, in the skiving apparatus 20 discussed above, having only one pair of rollers 24 and 26 which pair of rollers are disposed forward of the blade 34, once the trailing side of the belt 28 passes through the rollers 24 and 26 and out of engagement with the rollers, there is no longer any force driving the belt forward relative to the blade 34. To overcome this problem, the additional, rear pair of rollers 124a and 124b in the preferred embodiment skiving apparatus 120 which define a rear nip rearward of the blade 134 remain clamped about the belt 128 continue to drive the carriage 122 relative to the belt 128 after the trailing side 129 of the belt has passed through the forward pair of rollers 124b and 126b to drive the trailing side 129 of the belt 128 past the blade 134. Hence, the provision of an additional rear nip rearward of the blade 134 allows the tool to finish off the cut at the trailing side 129 of the belt 128 and eliminates the problem of a small uncut portion remaining at the trailing side 129 of the belt 128.

A slightly modified version of the skiving apparatus 120 is illustrated in Figs. 20-23. The following description of Figs. 20-23 utilizes prime reference numerals for parts similar to those used with skiving apparatus 120. The slightly modified skiving apparatus 120' is best seen in Fig. 21. Similar to the skiving apparatus 120, there is a vertical wall 144'; however, the horizontal wall 146 and wheels 147 in skiving apparatus 120 have been removed from the skiving apparatus 120'.

In skiving apparatus 120', the portion of the belt 128 adjacent its end 125 is passed through a pair of nips defined by respective pairs of rear and forward driving and pressure rollers 124a' and 126a', and 124b' and 126b', with their operation substantially the same as in skiving apparatus 120 and therefore, will be described
only to the extent necessary for an understanding of the basic operation of skiving apparatus 120' and the modifications made to skiving apparatus 120. As before, the forward nip is defined by the forward driving and pressure rollers 124b' and 126b' disposed closely ahead of the blade 134' so that the belt 128 has an upper strip 131 sliced therefrom. By providing the rear driving roller 124a' and rear pressure roller 126a', the belt 128 continues to be pulled so that its trailing end 129 engages the blade 134' to ensure a complete cut of the strip 131 near the trailing side 129 thereof.

The driving rollers 124a' and 124b' are interconnected by a chain 121', as seen in FIG. 20. The rear drive roller 124a' is rotated through a ratchet arm 232 attached to the rear drive roller 124a' so that alternate cranking and ratcheting of the ratchet arm 232 causes both of the forward and rear pair of rollers to advance the cutting blade 134' relative to the belt 128.

In the skiving apparatus 120', a bracket 150' is provided for supporting the rear and forward driving rollers 124a' and 124b' for rotation about their fixed rotational axes. The rear drive roller 124a' has an elongated axle 156a' which extends through the vertical wall 144', with the portion extending through the vertical wall having a hex nut 230 mounted thereto as by a pin 231. A similar elongated axle 156b' and hex nut 230 can be associated with the forward drive roller 124b'. At the other end of the rear and forward drive rollers 124a' and 124b', the axles 156a' and 156b are each formed into stub axles which are received in bearings 158' of a roller engaging bar portion 154' of the support bracket 150'.

The vertical wall 144' has a recess 159' milled therein from the bottom surface thereof. The recess 159' contains therein the sprocket wheels 162' and hubs thereof which are mounted on the elongated axle 156a and 156b. By milling the recess 159' from the bottom surface
of the vertical wall 144', the sprocket wheels 162' and chain are protected, particularly by the end face 157' of the vertical wall 144'.

Similar to the skiving apparatus 120, the pressure rollers 126a' and 126b' are mounted for rotation on pivotal brackets 160', which are, in turn, mounted on respective pivot pins 161' extending horizontally forward from the vertical wall 144'. By use of actuating member 164', the belt clamping force can be simultaneously equally adjusted on the pairs of rollers 124a', 126a', and 124b', 126b'. As in the skiving apparatus 120, the actuating member 154' has a threaded shaft 166' which moves vertically through the wall 144' upon manual rotation of integral turning knob 173' of the actuating member 164'.

To adjust the vertical blade height and therefore the thickness of the belt strip 131 that will be cut, a blade adjusting means 196' is provided. Similar to blade adjusting means 196, blade adjusting means 196' includes a threaded shaft 196a' which extends into engagement with a blade carrying block for movement of the blade 134' in the vertical direction.

To adjust the width of the strip cut from the belt end, skiving apparatus 120' includes an adjustable fence 200' which is supported for movement to and from vertical wall 144'. The vertical wall 144' includes apertures 146' adjacent its rear and forward ends 148' and 149' for receipt of ends of guide legs 152' as by a friction fit therein. At the other end of the guide legs 152', the guide legs 152' are secured in apertures of the roller engaging bar portion 154'. In the forward surface of the roller engaging bar portion 154', countersunk recesses 248 are provided so that set screws 250 can be inserted therein to be threadably received in the guide legs 152'. Thus, the support bracket 150' is formed by the roller engaging bar portion 154' and the guide legs 152'.

The fence 200' is slidably supported on the guide legs 152' which extend through corresponding apertures 252 therein. The fence 200' also has a central aperture 254. The central aperture 254 is adapted to receive a guide rod 256 extending horizontally from the vertical wall 144'. The fence 200' includes rear and forward arcuate cutout portions 212' and 214' for accommodating the respective rear and forward drive rollers 124a and 124b and an intermediate cutout portion 216' for accommodating the blade 134'. The cutout portions 212', 214', and 216' allow free forward and rearward sliding movement of the fence 200' along the guide legs 152' and the guide rod 256.

For selective positioning of the fence 200 for a specific desired width of a belt cut, the fence 200' includes a centrally disposed cam lever 258 associated therewith, as best seen in FIG. 21. The cam lever 258 is pivotably attached to the fence 200' by pin 260 so that the cam lever 258 can be pivoted between a locked position and a release position. In the locked position, as shown in solid lines in FIG. 21, the cam lever 258 is in a horizontal position such that cam surface 262 thereof tightly engages a top surface 264 of the guide rod 256 to prevent the fence 200' from translating along the guide legs 152' and guide rod 256. To facilitate the locking action between the cam surface 262 and the top surface 264, the top surface can be provided with a series of notches (not shown) therein for receipt of the cam surface 262. To release the fence 200' for sliding movement towards and away from the vertical wall 144', the cam lever 258 is pivoted clockwise to a release position, as seen in dashed lines in FIG. 21, wherein the cam surface 262 is pivoted out of engagement with the top surface 264 of the guide rod 256, thereby allowing free translation of the fence 200' along the guide legs 152' and central guide rod 256. The guide rod 256 can be provided with indicia along its length on the top surface.
264 thereof, for example alongside each notch, to provide an indication of the exact width of the belt being cut. The foregoing disclosure of specific embodiments is intended to be illustrative of the broad concepts of the invention described herein.
What Is Claimed Is:

1. An apparatus for skiving a conveyor belt on a support surface to remove a portion of the conveyor belt from one of its faces, the apparatus comprising:
   a carriage for traveling along the support surface;
   a belt cutting blade mounted on said carriage for cutting away a portion of the face of the conveyor belt; and
   drive means on the carriage forming a nip with the belt inserted into the nip for driving the carriage relative to the belt to cut the portion from the belt thereby skiving the belt.

2. An apparatus for skiving a conveyor belt to remove a portion of a face of the conveyor belt transversely across the width of the conveyor belt, the apparatus comprising:
   a carriage;
   a driving roller rotatably mounted to said carriage;
   a pressure roller rotatably mounted to said carriage, said driving roller and said pressure roller defining a nip therebetween for receipt of the belt, the driving roller and pressure roller being movable to a clamped position in which the rollers clamp together about the belt to securely engage the belt;
   a belt skiving blade mounted to said carriage adjacent said rollers at a selective position at which the upper portion of the belt passed between the rollers passes above the blade and the lower portion of the belt passes below the blade;
   a non-slip surface on at least one of the rollers for engaging the belt; and
   means for rotating the driving roller with the non-slip surface traveling along a face of the belt and
with the cutting blade cutting away the portion of the belt from the remainder of the belt as the carriage is driven across the width of the belt.

3. A belt skiving apparatus for removing a narrow portion of a conveyor belt adjacent an upper face of the conveyor belt, to form a groove in the belt and reduce the thickness of the belt to an accurate, predetermined thickness at said groove as registered off the bottom face of the belt, the apparatus comprising:

   a carriage;

   a pair of upper and lower rollers mounted to said carriage and movable to an engaging position in which the rollers clamp together about the belt, one of the rollers being a pressure roller bearing against the lower face of the belt and pinching the belt against the upper rollers;

   a blade mounted to said carriage directly adjacent said exit side of the rollers, for removing said narrow portion of the conveyor belt adjacent its upper face simultaneous with said lower face of the belt bearing against said pressure roller; and

   a toothed surface on the upper roller for puncturing the upper face of the belt and for rolling across the upper face of the belt as the belt is skived.

4. A method for skiving a conveyor belt to remove a portion of the conveyor belt, the method comprising:

   providing a carriage having a driving roller rotatably mounted to said carriage and a pressure roller rotatably mounted to said carriage, and having a skiving blade adjacent the rollers;

   inserting a conveyor belt between said driving roller and said pressure roller and clamping the belt in a nip between the rollers; and
rotating the driving roller to drive the carriage with the belt tracking through the nip and the skiving blade cutting away the upper portion of the belt from the lower portion of the belt as the carriage is driven across the width of the belt.

5. An apparatus for skiving a conveyor belt on a support surface to remove a portion of the conveyor belt from one of its faces from a first, leading side of the belt to a second, trailing side of the belt, the apparatus comprising:

a carriage for traveling along the support surface;

a belt cutting blade mounted on said carriage, said belt cutting blade having a forward cutting edge for cutting away a portion of the face of the conveyor belt;

first drive means on the carriage forming a first nip forward of said belt cutting blade with the belt insertable into the first nip for driving the carriage relative to the belt from said leading side of the belt toward said trailing side of the belt, with the belt cutting blade cutting said portion from the belt during said driving of the carriage relative to the belt, with the belt passing out of engagement with the first drive means upon the trailing side of the belt passing through said first nip, leaving a small uncut portion adjacent the trailing side of the belt; and

second drive means on the carriage forming a second nip rearward of said belt cutting blade with the belt insertable into the second nip and the second drive means having means for continuing said driving of the carriage relative to the belt subsequent to said trailing side of the belt passing through said first nip, to drive the forward cutting edge of the cutting blade past said trailing side of the belt to cut said small uncut portion remaining adjacent the trailing side of the belt.
6. A conveyor belt skiving apparatus in accordance with Claim 5 in which the first drive means and second drive means comprise respective pairs of rollers between which the conveyor belt is received, with the respective pairs of rollers clamping together on either side of the belt to engage the belt.

7. A conveyor belt skiving apparatus in accordance with claim 6 in which one of the pairs of rollers includes a drive roller having a drive axle portion extending through the carriage distal from the blade.

8. A conveyor belt skiving apparatus in accordance with claim 7 including a hex nut mounted on the drive axle portion distal from the blade and one of a pneumatically powered and a manually cranked ratchet arm having a socket configured to receive the hex nut therein with operation of the ratchet arm driving the drive axle portion through the socket and nut to rotate the drive roller.

9. A conveyor belt skiving apparatus in accordance with Claim 5 in which the carriage further comprises a fence defining a flat guiding surface for guiding the belt to maintain proper alignment of the belt with respect to the carriage.

10. A conveyor belt skiving apparatus in accordance with Claim 9 in which at least one of said respective pairs of rollers has means for biasing the belt toward the fence upon advancement of the belt relative to the rollers.

11. A conveyor belt skiving apparatus in accordance with Claim 10 in which the means for biasing
the belt toward the fence comprises spiralled threads on at least one of said rollers.

12. A conveyor belt skiving apparatus in accordance with Claim 9 in which the fence is repositionable in a direction normal to the forward driving of the carriage for selective variability of the width of the portion cut from the belt.

13. A conveyor belt skiving apparatus in accordance with Claim 12 in which the apparatus comprises means for selectively adjusting the vertical position of the horizontal portion of the blade for selective variation of the thickness of the portion cut from the belt.

14. A conveyor belt skiving apparatus in accordance with Claim 12 in which the apparatus further comprises a fence defining a flat guiding surface for guiding the belt to maintain proper alignment of the belt relative to the carriage, the fence being repositionable toward and away from the vertical portion of the belt cutting blade for selective variation of the width of the portion cut from the belt.

15. A conveyor belt skiving apparatus in accordance with claim 14 including a cam means for locking the guiding surface in a plurality of different positions relative to the vertical portion of the belt cutting blade for selective variation of the width of the portion cut from the belt.

16. A conveyor belt skiving apparatus in accordance with Claim 5 in which said cutting blade is generally J-shaped and defines a horizontal blade portion and a vertical blade portion.
17. An apparatus for skiving a conveyor belt on a support surface to remove a portion of the conveyor belt from one of its faces adjacent one of its ends, the apparatus comprising:

a carriage for traveling along the support surface;

a belt cutting blade mounted on said carriage for cutting away a portion of the face of the conveyor belt;

drive means on the carriage forming a nip with the belt inserted into the nip for driving the carriage relative to the belt to cut the portion from the belt thereby skiving the belt; and

an adjustable guiding surface on the carriage for guiding the end of the belt to maintain proper alignment of the belt with respect to the carriage and being selectively adjustable in a direction for selective variability of the width of the portion cut from the belt.

18. A conveyor belt skiving apparatus in accordance with Claim 17 in which said cutting blade is a generally J-shaped blade having a horizontal blade portion and a vertical blade portion, and said cutting blade is vertically repositionable for selective variability of the thickness of the portion cut from the belt.

19. A conveyor belt skiving apparatus in accordance with claim 17 in which the drive means includes a pair of rollers between which the conveyor belt is received, with the rollers clamping together on either side of the belt to engage the belt and one of the rollers having a drive axle portion extending through the carriage distal from the blade.

20. A conveyor belt skiving apparatus in accordance with claim 19 including a polygonal nut mounted on the drive axle portion distal from the blade.
and one of a pneumatically powered and a manually cranked ratchet arm having a socket configured to receive the polygonal nut therein with operation of the ratchet arm driving the drive axle portion through the socket and nut to rotate the drive roller.

21. A conveyor belt skiving apparatus in accordance with claim 17 including a cam means for locking the adjustable guiding surface in a plurality of different positions relative to the carriage for selective variability of the width of the portion cut from the belt.

22. A conveyor belt skiving apparatus in accordance with claim 21 wherein the cam means includes a lever associated with the guiding surface and a guide rod on which the guiding surface translates to the plurality of different positions with the lever having a cam surface thereon cooperating with the guide rod, the lever having a locked position wherein the guiding surface is fixed relative to the carriage in one of the plurality of different positions and a release position wherein the guiding surface can translate on the guide rod to another one of the plurality of different positions.

23. A method for skiving a conveyor belt to remove a portion of the conveyor belt, the method comprising:

   providing a carriage having first and second sets of driving and pressure rollers rotatably mounted to said carriage, and having a skiving blade adjacent one set of the rollers;

   inserting a conveyor belt between said first and second sets of driving and pressure rollers and clamping the belt in a nip between the respective rollers;
rotating the driving rollers of both sets and driving the carriage with the belt tracking through the nip and cutting away the upper portion of the belt from the lower portion of the belt with the skiving blade as the carriage is driven across the width of the belt; and using one set of rollers to drive the carriage when the other set is no longer driving to finish the last remaining cut of the upper portion from the lower portion.

24. A method in accordance with Claim 23 including the step of setting the width of cut relative by adjusting a guide on the carriage to a position and then sliding the guide along the belt edge as the carriage travels along the belt end.
# INTERNATIONAL SEARCH REPORT

## A. CLASSIFICATION OF SUBJECT MATTER

**IPC(6)**: B26D 3/28  
**US CL**: 83/352, 353, 435.1, 436, 870  
According to International Patent Classification (IPC) or to both national classification and IPC

## B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

**U.S.**: 83/352, 353, 423, 425, 435.1, 436, 633, 640, 856, 857, 858, 870, 874, 875

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic database consulted during the international search (name of data base and, where practicable, search terms used)

## C. DOCUMENTS CONSIDERED TO BE RELEVANT

<table>
<thead>
<tr>
<th>Category*</th>
<th>Citation of document, with indication, where appropriate, of the relevant passages</th>
<th>Relevant to claim No.</th>
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<tbody>
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Further documents are listed in the continuation of Box C.  
See patent family annex.

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  * **O**
    - document referring to an oral disclosure, use, exhibition or other means
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    - document published prior to the international filing date but later than the priority date claimed

**Date of the actual completion of the international search**: 12 NOVEMBER 1995  
**Date of mailing of the international search report**: 08 DEC 1995

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