United States Patent [19]

Gustmann et al.

[45] Sept. 11, 1973

[54]	UPPER TRANSPORT DEVICE FOR SEWING	3,570,426 3/1971 Kun 3,570,427 3/1971 Wer	
	MACHINES	3,614,934 10/1971 Wei	
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[22]	Filed: June 5, 1972		
[21]	Appl. No.: 259,424	[57] ABS	
[30]	Foreign Application Priority Data	The means for feeding the	
	June 19, 1971 Germany G 71 23 663.0	sewing machine (especia chine) comprises an upper	
[52]	U.S. Cl 112/212	rial pressure foot. Acco	
[51]	Int. Cl	arate spring means, inde	
[58]	Field of Search 112/70, 203, 207,	provided to apply pres	
	112/212	foot, and the material pr	
[56]	References Cited	of the moving parts car sirable inertia effects car	
	UNITED STATES PATENTS	the machine.	
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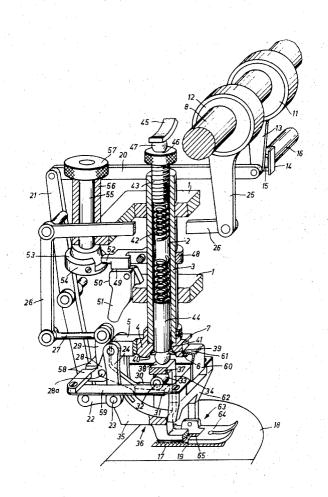
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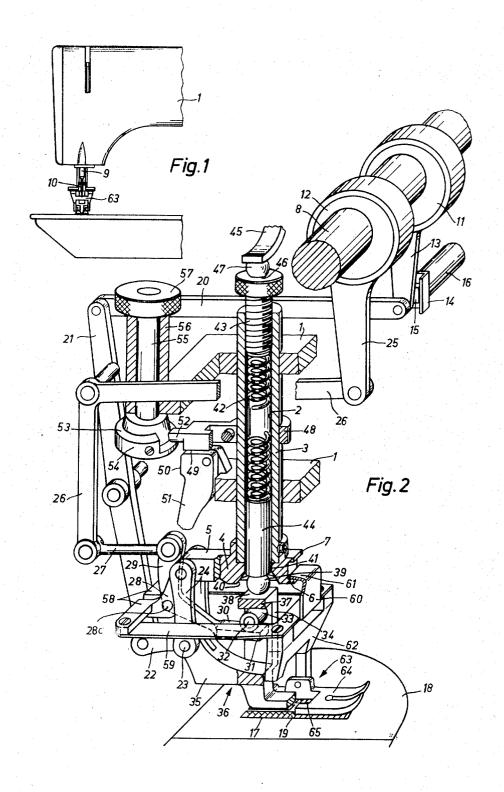
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Drawing Figures





UPPER TRANSPORT DEVICE FOR SEWING **MACHINES**

The invention concerns an upper-transport device for a sewing machine with a spring operated stopping 5 of the material pusher foot and the upper transport foot carried on the material pusher rod.

By known arrangements of the art, undesired vibrations occur at high operating speeds which are proon the sewed goods or on the lower material pusher and through the inertia of the moving parts. Especially with high speed sewing machines, there is not guaranteed, under these conditions, an orderly performance from the upper transporting foot as well as the necessary contact between the material presser foot and the sewing goods. As a result, the material to be sewed is not equally transported.

For the reduction of the moving mass, upper transport arrangements are already known in which the combination between the material pressure foot and the upper transporting foot is developed, so that the pressure foot no longer lifts off the to-be-sewed material, but only releases it. Thereby, the moving mass is indeed somewhat reduced, however, the resultant inertia of the material pressure system, which the lifting and lowering motion of the lower material pusher brings forth, is not essentially reduced.

mass of an upper transport arrangement and to attain this object by supporting the material pressure rod from a parallel, adjustably attached stop provided on the sewing machine and by arranging for both the material pressure foot and the upper transporting foot op- 35 posite the material pressure rod to move vertically at any given time by an allotted amount independently of each other, and to remain under the influence of a suit-

In the attached drawing which illustrates, by way of 40 example, a constructional form of the invention;

FIG. 1 is a partial front view of a sewing machine with the upper transport device of the invention, and

FIG. 2 is an enlarged prospective view, partially in section, of the upper transport device.

The material presser rod 3, having a bore 2, is supported in housing 1 in the sewing machine, and to the lower end thereof the bracket 4 is fastened with screws. The bracket 4 is equipped with an arm 5 projecting behind the rod 3 (with respect to the front of the ma- 50 chine), and has at its forward end two parallel stops 6 and 7, one above the other.

In the housing 1 of the sewing machine, there is, furthermore, supported in the known manner, the main driving shaft 8, which through a driving element (not shown) drives the needle rod 9 (FIG. 1), containing the needle 10. Resides, there is attached to the main shaft 8 (FIG. 2) a first eccentric 11 and a second eccentric 12.

The ring follower of eccentric 11 is connected to the eccentric rod 13 on which is encamped the slide block 15 held in the rocker arm guide 14. The direction of the guideway 14 is determined by the adjustable shaft 16 which is supported in the housing with a hand lever (not shown), projecting from the housing. With the hand lever, the size and direction of the feed movement of the material advancing means 19, projecting through

the slit 17 of the stitching plate 18 is determined in the known way.

As a result of the enforced guiding of the slider block 15 in the guide 14 and the movement of the slide block 15 resulting from the eccentric rod 13, a horizontal or sideward movement is transmitted to the rod 13 which depends on the inclination of the guide 14. A link 20 transmits the horizontal or side movement of rod 13 to a double lever 21 which is supported in the housing, duced through the reaction of the upper transport foot 10 and thence to a link 22 mounted on pivot 23 which is carried by a pair of links 24 on the arm 5 of bracket 4.

> The eccentric 12 drives the eccentric rod 25 which, through bell crank 26 (supported in housing 1) and a link 27 operates a lever 29, which in turn, is supported 15 from a pivot 28a on a forked-like extension 28 of the arm 5.

> The free arm 31 of lever 29 has a smooth upper surface 30, which stretches under the pin 32 rotatably supporting roller 33. The pin 32 is attached in the yoke-20 like shaft 34 (partly cut away) mounted on arm 35, which is also supported from pivot 23 as part of the upper transporting foot 36.

The horizontal wall portion 37 of the yoke-like shaft 34 has a smooth pressure surface 38 adjacent the lower end of bracket 4 on which the ball 39, extending from the lower flared surface 40 of bore 41 in bracket 4, the flaring of surface 40 being adapted to accommodate the ball 39.

The invention seeks to essentially reduce the moving 30 ing 2 in the material pressure rod 3. In said boring 2, The bore 41, in bracket 4, is an extension of the bora spring 42 is provided which contacts adjacent the upper end of the rod 3, an adjusting screw 43 threadedly connected to the rod 3 and which spring contacts at its other end, a bolt 44 which is guided in the borings 2 and 41 and presses against ball 39.

The leaf spring 45, which is attached to the housing in a known way, presses with its end against ball 47, held in the cup shaped recess 46 of the adjusting screw 43. By screwing adjusting screw 43 in or out the pressure of leaf spring 45 can be changed.

A lifter 48 is attached to the material pressure rod 3 and has a stop surface 49 cooperating with stop 50 of the pivotable hand lever 51 which is attached to the housing 1, as shown.

The arm 52, which is rigidly connected to or an integral part of the lifter member 48, cooperates with the helically rising curved track 53 of the fitting 54. Said fitting 54 is carried at the end of shaft 55 mounted in bearing eye 56 of housing 1, and said shaft 55 carries a rotatable knob 57 at its outer end.

The extension 28 of arm 5 comprises two supporting arms 58 extending at right angles to the extension 28 and to the ends of which a yoke-formed spring 59 is attached. This spring carries at its front side an upwardly extending, curved clip or claw 60, the free end 61 of which is inserted between stops 6 and 7. On the front side of spring yoke 59 is attached the shaft 62 of the material presser foot 63 whose base 64 presses the goods to be sewed against the advancing means 19. The base 64 is provided with a slot 65, through which the upper transport foot 36 can bring the sewed goods and the material advancing means 19 into contact.

In operation, the spring 45 holds the arm 52 of the material presser rod 3 on the curved track of the fitting 54. Through rotation of the knob 57, the support bearing for the material presser rod 3 can be adjusted to the strength (thickness) of the goods to be sewed. With it, the material presser rod 3 and its attached bracket 4, will be effectively lowered so far, that the base 64 of the material-presser-foot 63 rests on the goods to be sewed under the action of spring 59 when the advancing means 19 is lowered below the stitch plate 18 and so 5 that the end of clip 60 is lifted straight off of stop 6. The material presser foot 63 will no longer be pressed on the goods to be sewed by the pressure of spring 50 when the pressure of leaf spring 45 presses heavily on the fitting 54 and is not transmitted to the material 10 pressure foot 63.

The apparatus works as follows:

Rotation of the main shaft 8 propels the eccentrics 11 and 12 together with the sewing mechanism of the ma-

The eccentric 11 displaces the rod 13 with its slide block 15 in the guide 14. According to the position of the guide 14, which can be adjustably changed through shaft 16, the double lever 21 is more or less vigorously oscillated and a corresponding degree of feeding move- 20 ment is transmitted to the upper transport foot 36 through link 22.

The horizontal movement of upper transport foot 36 is made possible through contact between roller 33 and the surface 30, as well as between surface 38 and ball 25 39, and between bolt 44 and the roller bearing. The stop surface 40 limits the movement of ball 39 and the relative position of the upper surface 38 of the wall portion 37.

The eccentric 12 oscillates the bell crank lever 26 30 through rod 25 and thus oscillates the lever 29 through link 27. Thereby, the arm 31 of lever 29 is lifted and lowered. Upon lifting arm 31, the upper surface 30 thereof pushes the upper transport foot 36 upward by means of the roller 33. The lifting is carried out agains 35 the action of spring 42, the loading pressure of which is applied to ball 39. As soon as the upper transport foot 36 has lifted, the sewed goods accordingly is loaded only by the pressure of spring 59 acting on material pressure foot 63. The lower material advancing 40 means 19 has then dropped below the upper surface of stitch plate 18.

By lowering of arm 31, the roller 33 is released from upper surface 30 whereupon the upper transport foot 36 touches down on the goods being sewed through the 45 action of spring 42. Simultaneously, the teeth of the material advancing means 19 move through the open slit 17 of stitch plate 18 above the surface of the stitch plate to lift the material pressure foot 63. Since the distance between the two stops 6 and 7 is greater than the 50 height of the lift of the material advancing means 19 above the surface of stitch plate 18, the spring 59 takes up all of the lifting movement of the material pressure foot 63. The material pressure rod 3 remains in its lower position.

At first, when the material pressure foot 63 is lifted, so as to pass a certain thickness of sewing material, whereupon the end 61 of clip 60 is supported against the upper stop 7, the clip 60 lifts the material presser rod 3 and the pressure of leaf spring 45 acts now addi- 60 tionally through the material pressure rod on the material pressure foot 63.

The upper transport system of the invention has, accordingly, a stroke limit for the lower material advancing means 19 operating independently from the springi- 65 pressure and having an arm adapted to ride on said cam ness of the upper transport foot 36 and the material presser foot 63. By this means, in the normal sewing

process, only the slight mass of the two feet is required to be accelerated and retarded. When, on the other hand, the change of material in the sewing process comes up - when additional pressure will be necessary the system works as the known upper transport equipment.

For the lifting of the material pressure foot 63, the hand lever 51 is rotated. Thereby the stop 50 of the lift member 48 and the material pressure rod 3 is lifted. Simultaneously, the stop 6 moves upwardly and takes along the clip 60 so that the material presser foot 63 is lifted from the sewing goods. As a result of the pivoting, the upper transport foot 36 will, as well, be correspondingly lifted through the link pair 24 and also through the pivoting of lever 29 on carrier 4.

I claim:

1. An upper transport arrangement for a sewing machine having a main drive shaft, a casing, a material pressure foot and an upper transport foot and drive mechanisms to operate said pressure foot and said upper transport foot from the main drive shaft compris-

a vertically moveable material pressure rod mounted in the casing of the machine,

a first spring disposed in said casing acting to urge said material pressure rod downwardly,

a stop means adjustably mounted in the casing of the machine for supporting the material pressure rod with respect to said first spring,

means connecting the material pressure rod with the material pressure foot constructed and arranged to make possible a vertical movement between the material pressure rod and the material pressure foot, said means including a second spring means associated with said pressure rod normally urging the material pressure foot against the goods to be sewed.

a pair of stop means mounted on the material pressure rod and cooperating lug means on the material pressure foot projecting into the space between said pair of stop means to limit the vertical movement of the material pressure foot with respect to the material pressure rod,

means connecting the material pressure rod and the upper transport foot constructed and arranged to make possible a vertical movement between the material pressure rod and the upper transport foot, said means including a third spring means associated with said pressure rod, independent of the second spring means and normally urging the upper transport foot against the goods to be sewed,

a lever linked with the material pressure rod for imparting a lifting and lowering movement to the upper transport foot against the action of the third spring means, roller means on the upper transport foot through which said lever operates,

and means linking said lever to the main drive shaft to impart an oscillating movement to the lever.

2. The transport means as claimed in claim 1, and including control means to adjust for the thickness of the material to be sewed comprising a rotatable shaft parallel to the material pressure rod carrying a curved spiral cam surface, and collar means attached to said material surface.