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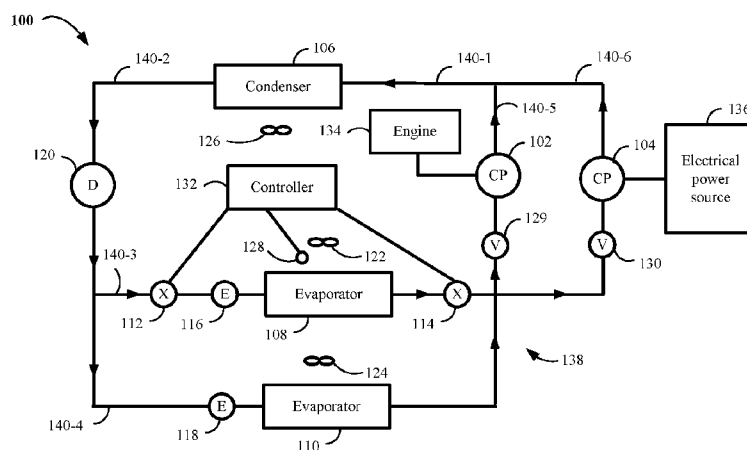
(54) **Title:** AIR CONDITIONING SYSTEM FOR USE IN VEHICLE

FIG. 1

- (57) **Abstract:** An air conditioning system for use in a vehicle includes at least one compressor, a condenser (106) disposed downstream of the at least one compressor and a plurality of evaporators disposed downstream of the condenser with first (108) and second (110) evaporators fluidly coupled to each other in parallel. The at least one compressor, the condenser (106), and the plurality of evaporators are fluidly connected by refrigerant lines (140) to form a refrigerant circuit (138). The air conditioning system also includes a plurality of shut-off valves, with one shut-off valve (112) installed at a refrigerant inlet and another shut-off valve (114) installed at a refrigerant outlet of the first evaporator (108). The shut-off valves are controlled to prevent undesired collection of refrigerant in the first evaporator (108).

AIR CONDITIONING SYSTEM FOR USE IN VEHICLE

FIELD OF THE INVENTION

[0001] The present invention generally relates to vehicle air conditioning systems, including, but not limited to, over-the-road and off-road vehicle air conditioning systems that can prevent undesired collection of refrigerant in one or more evaporators.

BACKGROUND

[0002] In response to the needs of the transportation industry and the regulations governing the amount of time that a driver may spend behind the wheel, many vehicles include a cab compartment and a sleeper compartment where the driver or passenger can rest. To create a comfortable environment for drivers and passengers, vehicle heating, ventilation, and air conditioning (HVAC) systems have been developed to provide conditioned air to the cab and sleeper compartments.

[0003] Such vehicle HVAC systems may include multiple evaporators, one for each compartment. For example, a system may include a cab evaporator (e.g., an evaporator associated with the cab compartment to cool the cab compartment) and a sleeper evaporator (e.g., an evaporator associated with the sleeper compartment to cool the sleeper compartment). Such systems may also include multiple compressors connected to different power sources so that the vehicle HVAC systems can be operated when the engine of the vehicle is on and when it is off.

[0004] To meet a specific cooling demand and reduce waste of compressed and condensed refrigerant, the systems may be operated to allow the compressed and condensed refrigerant to pass through one or more of the evaporators while restricting the condensed refrigerant from passing through other evaporator(s). For example, in a case where cooling in the sleeper compartment is not desired (e.g., the sleeper compartment is not occupied), the systems may be operated to restrict the refrigerant from passing through the sleeper evaporator to reduce waste of the compressed and condensed refrigerant on an unoccupied compartment. Similarly, in a case where cooling in the cab compartment is not desired, the systems may be operated to restrict the refrigerant from passing through the cab compartment.

[0005] While restricting the refrigerant from passing through the non-operating evaporator or evaporators (e.g., the sleeper evaporator when cooling in the sleeper compartment is not desired or the cab evaporator where cooling in the cab compartment is not desired), conventional systems cannot prevent the collection or accumulation of the refrigerant in the non-operating evaporator(s), in particular at the lower pressure side of the non-operating evaporator(s). The collection or accumulation of the refrigerant in the non-operating evaporator(s) reduces the effective amount of the refrigerant that should be used to cool the compartment(s) in need of cooling, making it difficult to meet (in some circumstances, only partially meet) the cooling demand. Consequently, the cooling capacity and overall efficiency of the HVAC systems are reduced, and the operational costs are increased.

[0006] Given the above background, there is a need in the art for air conditioning systems with enhanced cooling efficiency and capacity that provide conditioned air to multiple compartments and that prevent undesired collection of refrigerant in non-operating evaporators.

[0007] The information disclosed in this Background section is provided for an understanding of the general background of the invention and is not an acknowledgement or suggestion that this information forms part of the prior art already known to a person skilled in the art.

SUMMARY

[0008] Various aspects of the present invention provide air conditioning systems with enhanced cooling efficiency and capacity that provide conditioned air to multiple compartments and that prevent the undesired collection of refrigerant in non-operating evaporators.

[0009] In one embodiment, the present invention provides an air conditioning system for use in a vehicle having two or more compartments. The air conditioning system includes at least one compressor, a condenser disposed downstream of the at least one compressor, a plurality of evaporators disposed downstream of the condenser, a plurality of shut-off valves and refrigerant lines. The refrigerant lines fluidly connect the at least one compressor, the condenser, and the plurality of evaporators to form a refrigerant circuit for circulating the refrigerant.

[0010] In a preferred embodiment, the plurality of evaporators includes a first evaporator and a second evaporator fluidly coupled to each other in parallel. When implemented in the vehicle, the first evaporator is in thermal communication with a first compartment of the vehicle to cool the first compartment and the second evaporator is in thermal communication with a second compartment of the vehicle to cool the second compartment. The plurality of shut-off valves includes a first shut-off valve and a second shut-off valve. The first shut-off valve is installed at a refrigerant inlet of the first evaporator and the second shut-off valve is installed at a refrigerant outlet of the first evaporator. The first and second shut-off valves are controlled to prevent refrigerant from collecting in the first evaporator. In some embodiments, the first and second shut-off valves are controlled to prevent refrigerant from collecting in the first evaporator when airflow over or through the first evaporator is less than a first predetermined volume.

[0011] In some embodiments, the air conditioning system further includes a first sensor and a controller electrically coupled to the first sensor and the first and second shut-off valves. The first sensor is configured to perform one or more of the following: (i) measure temperature of the first evaporator and (ii) measure airflow passing over the first evaporator. The controller is configured to control the operation of the first and second shut-off valves in accordance with the measured temperature or the measured airflow or both. In some embodiments, the controller is configured to automatically close the first and second shut-off valves when the measured temperature of the first evaporator is lower than a first predetermined temperature, or when the measured airflow passing over the first evaporator is less than a first predetermined volume, and to automatically open the first and second shut-off valves when the measured temperature of the first evaporator exceeds the first predetermined temperature, or when the measured airflow passing over the first evaporator is equal to or greater than the first predetermined volume. In an embodiment, the method includes manually and selectively opening or closing the first and second shut-off valves in accordance with temperature of the first evaporator or airflow passing over the first evaporator.

[0012] In another embodiment, the present invention provides an air conditioning system for use in a vehicle having two or more compartments. The air conditioning system includes at least one compressor, a condenser disposed downstream of the at least one compressor, a plurality of evaporators disposed downstream of the condenser, a plurality of shut-off valves, refrigerant lines, a first sensor and a controller. The refrigerant lines fluidly connect the at

least one compressor, the condenser, and the plurality of evaporators to form a refrigerant circuit for circulating the refrigerant.

[0013] In one embodiment, the plurality of evaporators includes a first evaporator and a second evaporator fluidly coupled to each other in parallel. The plurality of shut-off valves includes a first shut-off valve and a second shut-off valve. The first shut-off valve is installed at a refrigerant inlet of the first evaporator and the second shut-off valve is installed at a refrigerant outlet of the first evaporator. The first sensor is configured to perform one or more of the following: (i) measure temperature of the first evaporator and (ii) measure airflow passing over the first evaporator. The controller is electrically coupled to the first sensor and configured to control the operation of the first and second shut-off valves in accordance with the measured temperature or the measured airflow or both. In some embodiments, the controller is configured to automatically close the first and second shut-off valves when the measured temperature of the first evaporator is lower than a first predetermined temperature, or when the measured airflow passing over the first evaporator is less than a first predetermined volume, and to automatically open the first and second shut-off valves when the measured temperature of the first evaporator exceeds the first predetermined temperature, or when the measured airflow passing over the first evaporator is equal to or greater than the first predetermined volume.

[0014] In an embodiment, the plurality of shut-off valves includes a third shut-off valve and a fourth shut-off valve. The third shut-off valve is installed at a refrigerant inlet of the second evaporator and the fourth shut-off valve is installed at a refrigerant outlet of the second evaporator to prevent refrigerant from collecting in the second evaporator. In some embodiments, the third shut-off valve and the fourth shut-off valve are controlled to prevent refrigerant from collecting in the second evaporator when airflow over the second evaporator is less than a second predetermined volume. In some embodiments, the system includes a second sensor configured to perform one or more of the following: (iii) measure temperature of the second evaporator and (iv) measure airflow passing over the second evaporator. In some embodiments, the controller is electrically coupled to the second sensor and configured to control the operation of the third and fourth shut-off valves in accordance with the measured temperature and the measured airflow. In some embodiments, the controller is configured to automatically close the third and fourth shut-off valves when the measured temperature of the second evaporator is lower than a second predetermined temperature, or when the measured airflow passing over the second evaporator is less than a second

predetermined volume, and to automatically open the third and fourth shut-off valves when the measured temperature of the second evaporator exceeds the second predetermined temperature, or when the measured airflow passing over the second evaporator is equal to or greater than the second predetermined volume.

[0015] The systems of the present invention have other features and advantages that will be apparent from or are set forth in more detail in the accompanying drawings, which are incorporated herein, and the following Detailed Description, which together serve to explain certain principles of the present invention.

BRIEF DESCRIPTION OF THE DRAWINGS

[0016] The accompanying drawings, which are incorporated into and constitute a part of this specification, illustrate one or more embodiments of the present application and, together with the detailed description, serve to explain the principles and implementations of the application.

[0017] FIG. 1 is a block diagram illustrating a first exemplary air conditioning system.

[0018] FIG. 2 is a block diagram illustrating a second exemplary air conditioning system.

[0019] FIG. 3 is a block diagram illustrating a third exemplary air conditioning system.

[0020] FIG. 4 is a block diagram illustrating a fourth exemplary air conditioning system.

[0021] FIG. 5 is a schematic diagram illustrating an implementation of an air conditioning system in a vehicle.

[0022] FIG. 6 is a schematic diagram illustrating an alternative implementation of an air conditioning system in a vehicle.

[0023] FIG. 7 is a schematic diagram illustrating another alternative implementation of an air conditioning system in a vehicle.

DETAILED DESCRIPTION

[0024] Reference will now be made in detail to implementations of the present application as illustrated in the accompanying drawings. The same reference indicators will be used throughout the drawings and the following detailed description to refer to the same or like parts. Those of ordinary skill in the art will realize that the following detailed description of the present application is illustrative only and is not intended to be in any way limiting.

Other embodiments of the present application will readily suggest themselves to such skilled persons having benefit of this disclosure.

[0025] In the interest of clarity, not all of the routine features of the implementations described herein are shown and described. It will, of course, be appreciated that in the development of any such actual implementation, numerous implementation-specific decisions must be made in order to achieve the developer's specific goals, such as compliance with application- and business-related constraints, and that these specific goals will vary from one implementation to another and from one developer to another. Moreover, it will be appreciated that such a development effort might be complex and time-consuming, but would nevertheless be a routine undertaking of engineering for those of ordinary skill in the art having the benefit of this disclosure.

[0026] Many modifications and variations of this disclosure can be made without departing from its spirit and scope, as will be apparent to those skilled in the art. The specific embodiments described herein are offered by way of example only, and the disclosure is to be limited only by the terms of the appended claims, along with the full scope of equivalents to which such claims are entitled.

[0027] Embodiments of the present invention are described in the context of air conditioning systems for use in vehicles, and in particular, in the context of air conditioning systems to cool different compartments or different spaces of an over-the-road or off-road vehicle. The vehicle can be a car, a van, a truck, a bus, a trailer, or other automobiles.

[0028] Generally, an air conditioning system of the present invention includes at least one compressor, a condenser, a plurality of evaporators and refrigerant lines. The refrigerant lines fluidly connect the compressor, condenser and evaporators to form a refrigerant circuit. The air conditioning system of the present invention also includes a plurality of shut-off valves disposed at the refrigerant circuit to prevent undesired collection of refrigerant in one or more of the evaporators, and thus enhance the cooling effect or capacity of the HVAC systems. In some embodiments, the air conditioning system of the present invention includes a controller electrically coupled to the shut-off valves to control the operation of the valves. In some embodiments, the controller is electrically coupled to other components of the air conditioning system (e.g., a compressor or a condenser or both) to control operation of these components.

[0029] By way of illustration, FIG. 1 depicts an air conditioning system (100) including a first compressor (102), a second compressor (104), a condenser (106), a first evaporator

(108), a second evaporator (110), a first shut-off valve (112) and a second shut-off valve (114). The first and second compressors, the condenser, the first and second evaporators, and the first and second valves are fluidly connected by refrigerant lines (104-1, 104-2, etc.) to form a refrigerant circuit (138).

[0030] In the illustrated embodiment, the first and second compressors (102, 104) are fluidly connected to each other in parallel by refrigerant lines (140-5, 140-6) and are configured for compressing a refrigerant. In some embodiments, the first compressor (102) is configured to connect to an internal combustion engine of the vehicle and is driven through a belt and pulley system by the internal combustion engine when the internal combustion engine is running. In some embodiments, the second compressor (104) is configured to operate when the internal combustion engine (134) of the vehicle is not running, for example, by connecting to an electrical power source (136) such as a battery. With the first and second compressors (102, 104) in place, the air conditioning system of the present invention can be operated when the engine is on as well as when the engine is off. In some embodiments, the first and second compressors (102, 104) are belt-driven compressors, electrically-driven compressors, or any other suitable compressors.

[0031] In some embodiments, an air conditioning system of the present invention has one compressor. For example, as illustrated in FIG. 2, the air conditioning system (200) of the present invention has one compressor (202). The compressor (202) can be a compressor driven by the engine (e.g., the same as or similar to the first compressor 102 in FIG. 1) or driven by a battery or other power sources (e.g., the second compressor 104 in FIG. 1).

[0032] Turning back to FIG. 1, the condenser (106) is disposed downstream of the first and second compressors (102, 104) and fluidly connected to the first and second compressors (102, 104) by a refrigerant line (140-1). The condenser (106) is configured to condense the refrigerant compressed by the first or second compressor or both. As used herein, the term “downstream” refers to a position along a refrigerant line in the direction of the refrigerant flow. As used herein, the term “upstream” refers to a position along a refrigerant line opposite to the direction of the refrigerant flow. For example, FIG. 1 illustrates the condenser (106) disposed downstream of the first and second compressors (102, 104) and upstream of the evaporator (108), where the directions of the refrigerant flow are indicated by the arrows.

[0033] The first and second evaporators (108, 110) are disposed downstream of the condenser (106) and fluidly connected to the condenser (106) by a refrigerant line (140-2). The first and second evaporators (108, 110) are fluidly coupled to each other in parallel via

refrigerant lines (140-3, 140-4) and configured to evaporate the condensed refrigerant. The first shut-off valve (112) is installed at the refrigerant inlet of the first evaporator (108), and the second shut-off valve (114) is installed at the refrigerant out of the first evaporator (108).

[0034] As used herein, the term “refrigerant inlet” refers to an inlet of a corresponding evaporator and a portion of a refrigerant line upstream of the corresponding evaporator. As used herein, the term “refrigerant outlet” refers to an outlet of a corresponding evaporator and a portion of a refrigerant line downstream of the corresponding evaporator. For example, refrigerant inlet of the first evaporator refers to the inlet of the first evaporator (108) and a portion of the refrigerant line (140-3) upstream of the first evaporator (108). Refrigerant outlet of the first evaporator (108) refers to the outlet of the first evaporator (108) and a portion of the refrigerant line (140-3) downstream of the first evaporator (108).

[0035] In some embodiments, the first evaporator (108) is in thermal communication with a first compartment and the second evaporator (110) is in thermal communication with a second compartment to cool the first and second compartments. As used herein, the term “in thermal communication” refers to one or more of the following: (i) the respective evaporator is mounted within a corresponding compartment to exchange heat with that compartment or with the air in that compartment, and (ii) the respective evaporator is coupled with a device (e.g., heat exchanger or air blower) which introduces conditioned air into that compartment.

[0036] In some embodiments, the first evaporator (108) is mounted in the first compartment and the second evaporator (110) is mounted in the second compartment. In some embodiments, the first compartment can be separated from the second compartment, for example, by a wall or other barrier such as a curtain. In some embodiments, the first compartment and the second compartment are different areas within the same space. In some embodiments, the first compartment is a cab compartment, a sleeper compartment, or any other compartment in a vehicle.

[0037] As an example, FIG. 5 illustrates an implementation of the air conditioning system (100) in a vehicle (502). The vehicle (502) has a cab compartment (504) where an operator or driver operates the vehicle and a sleeper compartment (506) where the operator or driver can rest. In the illustrated embodiment, first evaporator (108) is in thermal communication with the cab compartment (504), while second evaporator (110) is in thermal communication with the sleeper compartment (506).

[0038] When cooling is desired in both cab and sleeper compartments, first and second shut-off valves (112, 114) are opened, either manually or automatically, so that the condensed

refrigerant flows through both the first and second evaporators (108, 110) and provides cooling to both the cab and sleeper compartments. When cooling is only desired in the sleeper compartment (e.g., when the vehicle is parked and no one is in the cab compartment), the first and second shut-off valves (112, 114) are closed. Since the first and second shut-off valves (112, 114) are installed at both the refrigerant inlet and outlet of the first evaporator (108), closing the first and second shut-off valves (112, 114) prevents the refrigerant from entering the first evaporator (108) from both sides and thus prevents the refrigerant from collecting or accumulating in the first evaporator (108). As a result, the condensed refrigerant flows only through the second evaporator (110) and thus enhances the cooling effect of the second evaporator (110).

[0039] As another example, FIG. 6 illustrates an alternative implementation of the air conditioning system (100) of the present invention in the vehicle (502), where the first evaporator (108) is in thermal communication with the sleeper compartment (506) and the second evaporator (110) is in thermal communication with the cab compartment (504). In such an embodiment, when cooling is desired in both the cab and sleeper compartments, the first and second shut-off valves (112, 114) are opened, either manually or automatically, so that the condensed refrigerant flows through both the first and second evaporators (108, 110) and provides cooling to both cab and sleeper compartments. When cooling is only desired in the cab compartment (e.g., no one is in the sleeper compartment), the first and second shut-off valves (112, 114) are closed. Similar to the embodiment illustrated in FIG. 5, closing the first and second shut-off valves (112, 114) prevents the refrigerant from entering into the first evaporator (108) from both sides and thus prevents the refrigerant from collecting or accumulating in the first evaporator (108). As a result, the condensed refrigerant flows only through the second evaporator (110) and thus enhances the cooling effect of the second evaporator (110).

[0040] In some embodiments, the first and second shut-off valves are operated (e.g., opened or closed) manually, for example, by a driver of the vehicle who desires more cooling in a compartment/area or wants no cooling at all in the compartment/area. In some embodiments, the first or second shut-off valve or both are operated automatically, for example, by a controller (132). In some embodiments, operating the first and second shut-off valves depends on other parameters or operations of other components in the air conditioning system. For example, when airflow over the first evaporator is less than a first predetermined volume (e.g., indicating cooling is undesired in the compartment associated with the first

evaporator), the first and second shut-off valves are manually or automatically closed to prevent refrigerant from collecting in the first evaporator. The first predetermined volume can be preset or reset in accordance with the type of vehicle, compartments associated with the first evaporator, preference of the drivers/operators/passengers, ambient temperature or other parameters. In an example, the first predetermined volume is at most 75 Cubic Feet per Minute (CFM), indicating cooling is undesired in the compartment associated with the first evaporator.

[0041] In some embodiments, an air conditioning system of the present invention includes more than two shut-off valves. As an example, FIG. 3 illustrates an air conditioning system (300) having four shut-off valves, and FIG. 7 depicts an implementation of the air conditioning system (300) in the vehicle (502). In addition to the first and second shut-off valves (112, 114), the air conditioning system (300) includes a third shut-off valve (302) and a fourth shut-off valve (304). The third and fourth shut-off valves (302, 304) are installed at the refrigerant inlet and outlet of the second evaporator (110). Like the first and second shut-off valves (112, 114), the third and fourth shut-off valves (302, 304) can be opened/closed, manually or automatically. Similar to the first and second shut-off valves (112, 114), closing the third and fourth shut-off valves (302, 304) prevents the refrigerant from entering into the second evaporator (110) from both sides and thus prevents the refrigerant from collecting or accumulating in the second evaporator (110). As a result, the condensed refrigerant flows only through the first evaporator (108) and thus enhances the cooling effect of the second evaporator (108).

[0042] Like the first and second shut-off valves (112, 114), operation of the third and fourth shut-off valves (302, 304) can depend on other parameters or operations of other components in the air conditioning system. In some embodiments, the first, second, third and fourth shut-off valves are selectively and independently controlled. For example, when airflow over the second evaporator is less than a second predetermined volume (e.g., indicating cooling is undesired in the compartment associated with the second evaporator), the third and fourth shut-off valves are manually or automatically closed to prevent refrigerant from collecting in the second evaporator. Like the first predetermined volume, the second predetermined volume can be preset or reset in accordance with the type of vehicle, compartments associated with the second evaporator, desire of drivers/operators, or other parameters. The second predetermined volume can be the same as or different from the first predetermined volume. In an example, the second predetermined volume is at most 75 Cubic Feet per

Minute (CFM), indicating cooling is undesired in the compartment associated with the second evaporator.

[0043] Turning back to FIG. 1, in some embodiments, the air conditioning system (100) of the present invention includes a first sensor (128) and a controller (132) electrically coupled to the first sensor (128). In some embodiments, the control (132) is an intelligent power generation management controller described in U.S. Publication No. 2007/0131408 and U.S. Patent Nos. 7,591,143 and 8,453,722, all of which are expressly incorporated by reference in their entirety, and in particular with reference to intelligent power generation management controllers.

[0044] The first sensor (128) is configured to perform one or more of the following: (i) measure temperature of the first evaporator (108) and (ii) measure the airflow passing over the first evaporator (108). When the measured temperature is lower than a first predetermined temperature, or the measured airflow passing over the first evaporator (108) is less than the first predetermined volume (e.g., 75 CFM), or both, the controller (132) automatically closes or sends instruction to close the first and second shut-off valves (112, 114). When the measured temperature exceeds the first predetermined temperature or when the measured airflow passing over the first evaporator is equal to or greater than the first predetermined volume, the controller (132) automatically opens or sends instruction to open the first and second shut-off valves (112, 114). In some embodiments, the controller (132) is electrically coupled to one or more other components in the air conditioning system. For example, in one embodiment, the controller (132) is electrically coupled to the first compressor (102) or the second compressor (104) or both to automatically control the operation of the compressors in accordance with ambient temperature, operation of the engine, the cooling demand of the compartments of the vehicle, or other parameters.

[0045] In some embodiments, the air conditioning system (100) of the present invention includes a plurality of control valves to selectively restrict or permit flow of the refrigerant to the compressors. As an example, FIG. 1 illustrates the air conditioning system (100) having a first flow control valve (129) and a second flow control valve (130). The first flow control valve (129) is disposed at the refrigerant line (104-5) upstream of the first compressor (102) and configured to selectively restrict or permit flow of the refrigerant to the first compressor (102). The second flow control valve (130) is disposed at the refrigerant line (104-6) upstream of the second compressor (104) and configured to selectively restrict or permit flow

of the refrigerant to the second compressor (104). In some embodiments, the operation of the flow control valves is automatically controlled by the controller.

[0046] In some embodiments, the air conditioning system (100) of the present invention includes a plurality of metering devices to control flow of the refrigerant into the evaporators. As an example, FIG. 1 illustrates the air conditioning system (100) having a first metering device (116) and a second metering device (118). The first metering device (116) is disposed at the refrigerant line (104-3) between the first shut off valve (112) and the first evaporator (108) and configured for controlling flow rate of the refrigerant into the first evaporator (108). The second metering device (118) is disposed at the refrigerant line (104-4) upstream of the second evaporator (110) and configured for controlling flow rate of the refrigerant into the second evaporator (110). In some embodiments, the first and the second metering devices are thermal expansion valves. In some embodiments, the operation of the metering devices is automatically controlled by the controller.

[0047] In some embodiments, the air conditioning system (100) of the present invention includes a receiver/drier (120). The receiver/drier (120) is disposed at the refrigerant line (104-3) between the condenser (106) and the evaporators (108, 110). The receiver/drier (120) is configured to temporarily store the refrigerant, absorb moisture from the refrigerant, or both.

[0048] In some embodiments, the air conditioning system (100) of the present invention includes a plurality of air blowers to enhance the performance of some components in the air conditioning system. As an example, FIG. 1 illustrates the air conditioning system (100) having a first air blower (122), a second air blower (124) and a third air blower (126). The first air blower (122) is positioned proximate the first evaporator (108) and configured to blow air over the first evaporator (108). The air is cooled when passed over the first evaporator (108) and can be introduced into a compartment for cooling purpose. Similarly, the second air blower (124) is positioned proximate the second evaporator (110) and configured to blow air over the second evaporator (110). The air is cooled when passing the second evaporator (110) and can be introduced into another compartment or a different area of the same compartment for cooling purpose. The third air blower (126) is positioned proximate the condenser (106) and configured to blow air over the condenser (106) to cool it. When passing over the condenser (106), the air extracts the heat away from the condenser (106) and thus enhances the performance of the condenser (106).

[0049] Referring now to FIG. 3, in some embodiments, the air conditioning system (300) of the present invention includes more than one sensor for measuring airflows over the evaporators. For example, in addition to the first sensor (128), the air conditioning system (300), as illustrated, includes a second sensor (306) configured to perform one or more of the following: (iii) measure temperature of the second evaporator and (iv) measure the airflow passing over the second evaporator (116). In some embodiments, the controller (132) is electrically coupled to both the first and second sensors (128, 306) and configured to automatically control the first, second, third and fourth shut-off valves (112, 114, 302, 304) based on the measured temperatures or the measured airflows. In some embodiments, the controller (132) is configured to automatically perform or send instructions to perform one or more of the following: (i) close the first and second shut-off valves (112, 114) when the measured temperature of the first evaporator (108) is lower than a first predetermined temperature, or when the measured airflow passing over the first evaporator (108) is less than a first predetermined volume; (ii) open the first and second shut-off valves (112, 114) when the measured temperature of the first evaporator (108) exceeds the first predetermined temperature, or when the measured airflow passing over the first evaporator (108) is equal to or greater than the first predetermined volume; (iii) close the third and fourth shut-off valves (302, 304) when the measured temperature of the second evaporator (110) is lower than a second predetermined temperature, or when the measured airflow passing over the second evaporator (110) is less than a second predetermined volume; and (iv) open the third and fourth shut-off valves (302, 304) when the measured temperature of the second evaporator (110) exceeds the second predetermined temperature, or when the measured airflow passing over the second evaporator (110) is equal to or greater than the second predetermined volume. In some embodiments, one or more of the first, second, third and fourth shut-off valves (112, 114, 302, 304) are performed manually and selectively in accordance with the measured temperatures, airflows, or other factors.

[0050] In some embodiments, an air conditioning system of the present invention includes more than two evaporators. As an example, FIG. 4 illustrates an air conditioning system (400) having three evaporators. In addition to the first and second evaporator (108, 110), the air conditioning system (400) includes a third evaporator (402). The third evaporator (402) is fluidly coupled to the first and second evaporators (108, 110) in parallel by refrigerant lines (140-3, 140-4, 406). The third evaporator (402) is in thermal communication with a third compartment. The third compartment can be a compartment separated from the first and second compartments or an area in the first or second compartment. In some embodiments,

the air conditioning system (400) includes a third metering device (404) disposed at the refrigerant line (406) upstream of the third evaporator (402). The third metering device (404) is configured to control flow of the refrigerant into the third evaporator (402).

[0051] In some embodiments, the air conditioning system (400) includes the first and second shut-off valves (112, 114) installed at the refrigerant inlet and outlet of the first evaporator (108) to prevent undesired collection of the refrigerant in the first evaporator (108). In some embodiments, the air conditioning system (400) further includes the third and fourth shut-off valves, such as those (302, 304) illustrated in FIG. 3, installed at the refrigerant inlet and outlet of the second evaporator (110) to prevent undesired collection of the refrigerant in the second evaporator (110). In some embodiments, the air conditioning system (400) further includes another pair of shut-off valves installed at the refrigerant inlet and outlet of the third evaporator (402) to prevent undesired collection of the refrigerant in the third evaporator (402).

[0052] In some embodiments, an air conditioning system of the present invention is electrically, fluidly, thermally or mechanically coupled with other components, devices or systems. For example, an air conditioning system (e.g., 100, 200, 300, or 400) of the present invention is combined with a heating system to form a combined heating and cooling air conditioning system. The combination of an air conditioning system with a heating system can be achieved in a similar way as those described in U.S. Patent No. 8,517,087, which is expressly incorporated by reference in their entirety, and in particular with reference to the heating system.

[0053] As another example, an air conditioning system (e.g., 100, 200, 300, or 400) of the present invention is coupled to a vehicle ventilation module to provide conditioned fresh air to the desired compartment(s). The vehicle ventilation module can be the same as or similar to those described in U.S. Publication No. 2014/0262132, which is expressly incorporated by reference in their entirety, and in particular with reference to the vehicle ventilation module.

[0054] The terminology used herein is for the purpose of describing particular implementations only and is not intended to be limiting of the claims. As used in the description of the implementations and the appended claims, the singular forms “a”, “an” and “the” are intended to include the plural forms as well, unless the context clearly indicates otherwise. It will be understood that, although the terms “first,” “second,” etc. may be used herein to describe various elements, these elements should not be limited by these terms. These terms are only used to distinguish one element from another. For example, a first

evaporator could be termed a second evaporator, and, similarly, a second evaporator could be termed a first evaporator, without changing the meaning of the description, so long as all occurrences of the “first evaporator” are renamed consistently and all occurrences of the “second evaporator” are renamed consistently.

WHAT IS CLAIMED IS:

1. An air conditioning system for use in a vehicle having two or more compartments, the air conditioning system comprising:
 - at least one compressor for compressing refrigerant into a compressed refrigerant;
 - a condenser (106) disposed downstream of the at least one compressor for condensing the compressed refrigerant;
 - a plurality of evaporators disposed downstream of the condenser (106) for evaporating the condensed refrigerant, the plurality of evaporators including a first evaporator (108) and a second evaporator (110), wherein:
 - the first evaporator (108) and the second evaporator (110) are fluidly coupled to each other in parallel;
 - the first evaporator (108) is in thermal communication with a first compartment in the two or more compartments of the vehicle to cool the first compartment; and
 - the second evaporator (110) is in thermal communication with a second compartment in the two or more compartments of the vehicle to cool the second compartment;
 - a plurality of shut-off valves including a first shut-off valve (112) and a second shut-off valve (114), wherein the first shut-off valve (112) is installed at a refrigerant inlet of the first evaporator (108) and the second shut-off valve (114) is installed at a refrigerant outlet of the first evaporator (108) to prevent refrigerant from collecting in the first evaporator (108); and
 - refrigerant lines (140) fluidly connecting the at least one compressor, the condenser (106), and the plurality of evaporators to form a refrigerant circuit (138) for circulating the refrigerant.
2. The air conditioning system of claim 1, further comprising:
 - a first sensor (128) configured to perform one or more of the following: (i) measure temperature of the first evaporator and (ii) measure airflow passing over the first evaporator; and
 - a controller (132) electrically coupled to the first sensor and configured to:

automatically close the first and second shut-off valves (112, 114) when the measured temperature of the first evaporator is lower than a first predetermined temperature, or when the measured airflow passing over the first evaporator is less than a first predetermined volume, and automatically open the first and second shut-off valves (112, 114) when the measured temperature of the first evaporator exceeds the first predetermined temperature, or when the measured airflow passing over the first evaporator is equal to or greater than the first predetermined volume.

3. The air conditioning system of claim 2, wherein the controller (132) is electrically coupled to the at least one compressor and configured to automatically control operation of the at least one compressor in accordance with cooling demand of the two or more compartments of the vehicle.
4. The air conditioning system of any one of claims 1-3, further comprising:
manually and selectively opening or closing the first and second shut-off valves (112, 114) in accordance with temperature of the first evaporator or airflow passing over the first evaporator.
5. The air conditioning system of any one of claims 1-3, wherein the first evaporator (108) is mounted in the first compartment of the vehicle, wherein the first compartment is a cab compartment or a sleeper compartment.
6. The air conditioning system of any one of claims 1-3, wherein the plurality of evaporators further includes a third evaporator (402) fluidly coupled to the first and second evaporators (108, 110) in parallel and in thermal communication with a third compartment in the two or more compartments of the vehicle.
7. The air conditioning system of any one of claims 1-3, wherein the at least one compressor includes:
a first compressor (102) configured to connect to an internal combustion engine of the vehicle and driven by the internal combustion engine when the internal combustion engine is running; and
a second compressor (104) configured to operate when the internal combustion engine of the vehicle is not running,

wherein the first and second compressors (102, 104) are fluidly coupled to each other in parallel.

8. The air conditioning system of claim 7, wherein the second compressor (104) is an electrically driven compressor.
9. The air conditioning system of claim 7, further comprising:
 - a first flow control valve (129) disposed upstream of the first compressor (102) and configured to selectively restrict or permit flow of the refrigerant to the first compressor (102); and
 - a second flow control valve (130) disposed upstream of the second compressor (104) and configured to selectively restrict or permit flow of the refrigerant to the second compressor (104).
10. The air conditioning system of any one of claims 1-3, further comprising:
 - at least one flow control valve (129) disposed upstream of the at least one compressor and configured to selectively restrict or permit flow of the refrigerant to the at least one compressor.
11. The air conditioning system of any one of claims 1-3, further comprising:
 - a first metering device (116) disposed between the first shut off valve (112) and the first evaporator (108) and configured for controlling flow rate of the refrigerant into the first evaporator (108); and
 - a second metering device (118) disposed upstream of the second evaporator (110) and configured for controlling flow rate of the refrigerant into the second evaporator (110).
12. The air conditioning system of claim 11, wherein the first and the second metering devices (116, 118) are thermal expansion valves.
13. The air conditioning system of any one of claims 1-3, further comprising a receiver/drier (120) disposed between the condenser (106) and the plurality of evaporators (108, 110) and configured for performing one or more of the following: temporarily storing the refrigerant and absorbing moisture from the refrigerant.
14. The air conditioning system of claim 2, wherein the first predetermined volume is at most 75 Cubic Feet per Minute (CFM).

15. The air conditioning system of any one of claims 1-3, wherein the plurality of shut-off valves include a third shut-off valve (302) and a fourth shut-off valve (304), wherein the third shut-off valve (302) is installed at a refrigerant inlet of the second evaporator (110) and the fourth shut-off valve (304) is installed at a refrigerant outlet of the second evaporator (110) to prevent refrigerant from collecting in the second evaporator (110).

16. The air conditioning system of claim 15, further comprising:

a first sensor (128) configured to perform one or more of the following: (i) measure temperature of the first evaporator (108) and (ii) measure airflow passing over the first evaporator (108);

a second sensor (306) configured to perform one or more of the following: (iii) measure temperature of the second evaporator (110) and (iv) measure airflow passing over the second evaporator (110); and

a controller (132) electrically coupled to the first and second sensors (128, 306) and configured to:

automatically close the first and second shut-off valves (112, 114) when the measured temperature of the first evaporator (108) is lower than a first predetermined temperature, or when the measured airflow passing over the first evaporator (108) is less than a first predetermined volume, automatically open the first and second shut-off valves (112, 114) when the measured temperature of the first evaporator (108) exceeds the first predetermined temperature, or when the measured airflow passing over the first evaporator (108) is equal to or greater than the first predetermined volume,

automatically close the third and fourth shut-off valves (302, 304) when the measured temperature of the second evaporator (110) is lower than a second predetermined temperature, or when the measured airflow passing over the second evaporator (110) is less than a second predetermined volume, and

automatically open the third and fourth shut-off valves (302, 304) when the measured temperature of the second evaporator (110) exceeds the second predetermined temperature, or when the measured airflow passing over the second evaporator (110) is equal to or greater than the second predetermined volume.

17. The air conditioning system of claim 16, wherein the second predetermined volume is at most 75 Cubic Feet per Minute (CFM).
18. The air conditioning system of any one of claims 1-3, further comprising:
a first air blower (122) positioned proximate the first evaporator (108) and configured to blow air over the first evaporator (108);
a second air blower (124) positioned proximate the second evaporator (110) and configured to blow air over the second evaporator (110), and
a third air blower (126) positioned proximate the condenser (106) and configured to blow air over the condenser (106).
19. An air conditioning system for use in a vehicle, the air conditioning system comprising:
at least one compressor for compressing a refrigerant into a compressed refrigerant;
a condenser (106) disposed downstream of the at least one compressor for condensing the compressed refrigerant;
a plurality of evaporators disposed downstream of the condenser for evaporating the condensed refrigerant, the plurality of evaporators including a first evaporator (108) and a second evaporator (110), wherein the first evaporator (108) and the second evaporator (110) are fluidly coupled to each other in parallel;
a plurality of shut-off valves including a first shut-off valve (112) and a second shut-off valve (114), wherein the first shut-off valve (112) is installed at a refrigerant inlet of the first evaporator (108) and the second shut-off valve (114) is installed at a refrigerant outlet of the first evaporator (108) to prevent refrigerant from collecting in the first evaporator (108) when airflow over the first evaporator (108) is less than a first predetermined volume;
refrigerant lines (140) fluidly connecting the at least one compressor, the condenser (106), and the plurality of evaporators to form a refrigerant circuit for circulating the refrigerant;
a first sensor (128) configured to perform one or more of the following: (i) measure temperature of the first evaporator (108) and (ii) measure airflow passing over the first evaporator (108); and
a controller (132) electrically coupled to the first sensor (128) and configured to:
automatically close the first and second shut-off valves (112, 114) when the measured temperature of the first evaporator (108) is lower than a first

predetermined temperature, or when the measured airflow passing over the first evaporator (108) is less than a first predetermined volume, and automatically open the first and second shut-off valves (112, 114) when the measured temperature of the first evaporator (108) exceeds the first predetermined temperature, or when the measured airflow passing over the first evaporator (108) is equal to or greater than the first predetermined volume.

20. The air conditioning system of claim 19, further comprising:
the first evaporator (108) is in thermal communication with a cab compartment of the vehicle to cool the cab compartment; and
the second evaporator (110) is in thermal communication with a sleeper compartment of the vehicle to cool the sleeper compartment.
21. The air conditioning system of any one of claims 19-20, wherein the at least one compressor includes:
a first compressor (102) configured to connect to an internal combustion engine of the vehicle and driven by the internal combustion engine when the internal combustion engine is running; and
a second compressor (104) configured to operate when the internal combustion engine of the vehicle is not running,
wherein the first and second compressors (102, 104) are fluidly coupled to each other in parallel.

AMENDED CLAIMS

received by the International Bureau on 19 May 2017 (19.05.2017)

WHAT IS CLAIMED IS:

1. An air conditioning system for use in a vehicle having two or more compartments, the air conditioning system comprising:

at least one compressor for compressing refrigerant into a compressed refrigerant;

a condenser (106) disposed downstream of the at least one compressor for condensing the compressed refrigerant;

a plurality of evaporators disposed downstream of the condenser (106) for evaporating the condensed refrigerant, the plurality of evaporators including a first evaporator (108) and a second evaporator (110), wherein:
the first evaporator (108) and the second evaporator (110) are fluidly coupled to each other in parallel;

the first evaporator (108) is in thermal communication with a first compartment in the two or more compartments of the vehicle to cool the first compartment; and

the second evaporator (110) is in thermal communication with a second compartment in the two or more compartments of the vehicle to cool the second compartment;

a plurality of shut-off valves including a first shut-off valve (112) and a second shut-off valve (114), wherein the first shut-off valve (112) is installed at a refrigerant inlet of the first evaporator (108) and the second shut-off valve (112) is installed at a refrigerant outlet of the first evaporator (108) to prevent refrigerant from collecting in the first evaporator (108);

refrigerant lines (140) fluidly connecting the at least one compressor, the condenser (106), and the plurality of evaporators to form a refrigerant circuit (138) for circulating the refrigerant;

a first sensor (128) configured to measure airflow passing over the first evaporator; and

a controller (132) electrically coupled to the first sensor and configured to:

automatically close the first and second shut-off valves (112, 114) when the measured airflow passing over the first evaporator is less than a first predetermined volume, and

automatically open the first and second shut-off valves (112, 114) when the measured airflow passing over the first evaporator is equal to or greater than the first predetermined volume.

2. The air conditioning system of claim 1, wherein the first sensor (128) is further configured to measure temperature of the first evaporator.
3. The air conditioning system of claim 1, wherein the controller (132) is electrically coupled to the at least one compressor and configured to automatically control operation of the at least one compressor in accordance with cooling demand of the two or more compartments of the vehicle.
4. The air conditioning system of any one of claims 1-3, further comprising manually and selectively opening or closing the first and second shut-off valves (112, 114) in accordance with temperature of the first evaporator or airflow passing over the first evaporator.
5. The air conditioning system of any one of claims 1-3, wherein the first evaporator (108) is mounted in the first compartment of the vehicle, wherein the first compartment is a cab compartment or a sleeper compartment.
6. The air conditioning system of any one of claims 1-3, wherein the plurality of evaporators further includes a third evaporator (402) fluidly coupled to the first and second evaporators (108, 110) in parallel and in thermal communication with a third compartment in the two or more compartments of the vehicle.
7. The air conditioning system of any one of claims 1-3, wherein the at least one compressor includes:
 - a first compressor (102) configured to connect to an internal combustion engine of the vehicle and driven by the internal combustion engine when the internal combustion engine is running; and
 - a second compressor (104) configured to operate when the internal combustion engine of the vehicle is not running,wherein the first and second compressors (102, 104) are fluidly coupled to each other in parallel.
8. The air conditioning system of claim 7, wherein the second compressor (104) is an electrically driven compressor.
9. The air conditioning system of claim 7, further comprising:

- a first flow control valve (129) disposed upstream of the first compressor (102) and configured to selectively restrict or permit flow of the refrigerant to the first compressor (102); and
- a second flow control valve (130) disposed upstream of the second compressor (104) and configured to selectively restrict or permit flow of the refrigerant to the second compressor (104).
10. The air conditioning system of any one of claims 1-3, further comprising:
at least one flow control valve (129) disposed upstream of the at least one compressor and configured to selectively restrict or permit flow of the refrigerant to the at least one compressor.
11. The air conditioning system of any one of claims 1-3, further comprising:
a first metering device (116) disposed between the first shut off valve (112) and the first evaporator (108) and configured for controlling flow rate of the refrigerant into the first evaporator (108); and
a second metering device (118) disposed upstream of the second evaporator (110) and configured for controlling flow rate of the refrigerant into the second evaporator (110).
12. The air conditioning system of claim 11, wherein the first and the second metering devices (116, 118) are thermal expansion valves.
13. The air conditioning system of any one of claims 1-3, further comprising a receiver/drier (120) disposed between the condenser (106) and the plurality of evaporators (108, 110) and configured for performing one or more of the following: temporarily storing the refrigerant and absorbing moisture from the refrigerant.
14. The air conditioning system of claim 2, wherein the first predetermined volume is at most 75 Cubic Feet per Minute (CFM).
15. The air conditioning system of any one of claims 1-3, wherein the plurality of shut-off valves include a third shut-off valve (302) and a fourth shut-off valve (304), wherein the third shut-off valve (302) is installed at a refrigerant inlet of the second evaporator (110) and the fourth shut-off valve (304) is installed at a refrigerant outlet of the second evaporator (110) to prevent refrigerant from collecting in the second evaporator (110).

16. The air conditioning system of claim 15, further comprising:
a second sensor (306) configured to perform one or more of the following: (i) measure temperature of the second evaporator (110) and (ii) measure airflow passing over the second evaporator (110); and
wherein the controller (132) is electrically coupled to the second sensor and configured to:
automatically close the first and second shut-off valves (112, 114) when the measured temperature of the first evaporator (108) is lower than a first predetermined temperature, or when the measured airflow passing over the first evaporator (108) is less than a first predetermined volume,
automatically open the first and second shut-off valves (112, 114) when the measured temperature of the first evaporator (108) exceeds the first predetermined temperature, or when the measured airflow passing over the first evaporator (108) is equal to or greater than the first predetermined volume,
automatically close the third and fourth shut-off valves (302, 304) when the measured temperature of the second evaporator (110) is lower than a second predetermined temperature, or when the measured airflow passing over the second evaporator (110) is less than a second predetermined volume, and
automatically open the third and fourth shut-off valves (302, 304) when the measured temperature of the second evaporator (110) exceeds the second predetermined temperature, or when the measured airflow passing over the second evaporator (110) is equal to or greater than the second predetermined volume.
17. The air conditioning system of claim 16, wherein the second predetermined volume is at most 75 Cubic Feet per Minute (CFM).
18. The air conditioning system of any one of claims 1-3, further comprising:
a first air blower (122) positioned proximate the first evaporator (108) and configured to blow air over the first evaporator (108);
a second air blower (124) positioned proximate the second evaporator (110) and configured to blow air over the second evaporator (110), and

- a third air blower (126) positioned proximate the condenser (106) and configured to blow air over the condenser (106).
19. An air conditioning system for use in a vehicle, the air conditioning system comprising:
- at least one compressor for compressing a refrigerant into a compressed refrigerant;
 - a condenser (106) disposed downstream of the at least one compressor for condensing the compressed refrigerant;
 - a plurality of evaporators disposed downstream of the condenser for evaporating the condensed refrigerant, the plurality of evaporators including a first evaporator (108) and a second evaporator (110), wherein the first evaporator (108) and the second evaporator (110) are fluidly coupled to each other in parallel;
 - a plurality of shut-off valves including a first shut-off valve (112) and a second shut-off valve (114), wherein the first shut-off valve (112) is installed at a refrigerant inlet of the first evaporator (108) and the second shut-off valve (114) is installed at a refrigerant outlet of the first evaporator (108) to prevent refrigerant from collecting in the first evaporator (108) when airflow over the first evaporator (108) is less than a first predetermined volume;
 - refrigerant lines (140) fluidly connecting the at least one compressor, the condenser (106), and the plurality of evaporators to form a refrigerant circuit for circulating the refrigerant;
 - a first sensor (128) configured to perform one or more of the following: (i) measure temperature of the first evaporator (108) and (ii) measure airflow passing over the first evaporator (108); and
 - a controller (132) electrically coupled to the first sensor (128) and configured to:
 - automatically close the first and second shut-off valves (112, 114) when the measured airflow passing over the first evaporator (108) is less than a first predetermined volume, and
 - automatically open the first and second shut-off valves (112, 114) when the measured airflow passing over the first evaporator (108) is equal to or greater than the first predetermined volume.
20. The air conditioning system of claim 19, further comprising:
- the first evaporator (108) is in thermal communication with a cab compartment of the vehicle to cool the cab compartment; and

the second evaporator (110) is in thermal communication with a sleeper compartment of the vehicle to cool the sleeper compartment.

21. The air conditioning system of any one of claims 19-20, wherein the at least one compressor includes:

a first compressor (102) configured to connect to an internal combustion engine of the vehicle and driven by the internal combustion engine when the internal combustion engine is running; and

a second compressor (104) configured to operate when the internal combustion engine of the vehicle is not running,

wherein the first and second compressors (102, 104) are fluidly coupled to each other in parallel.

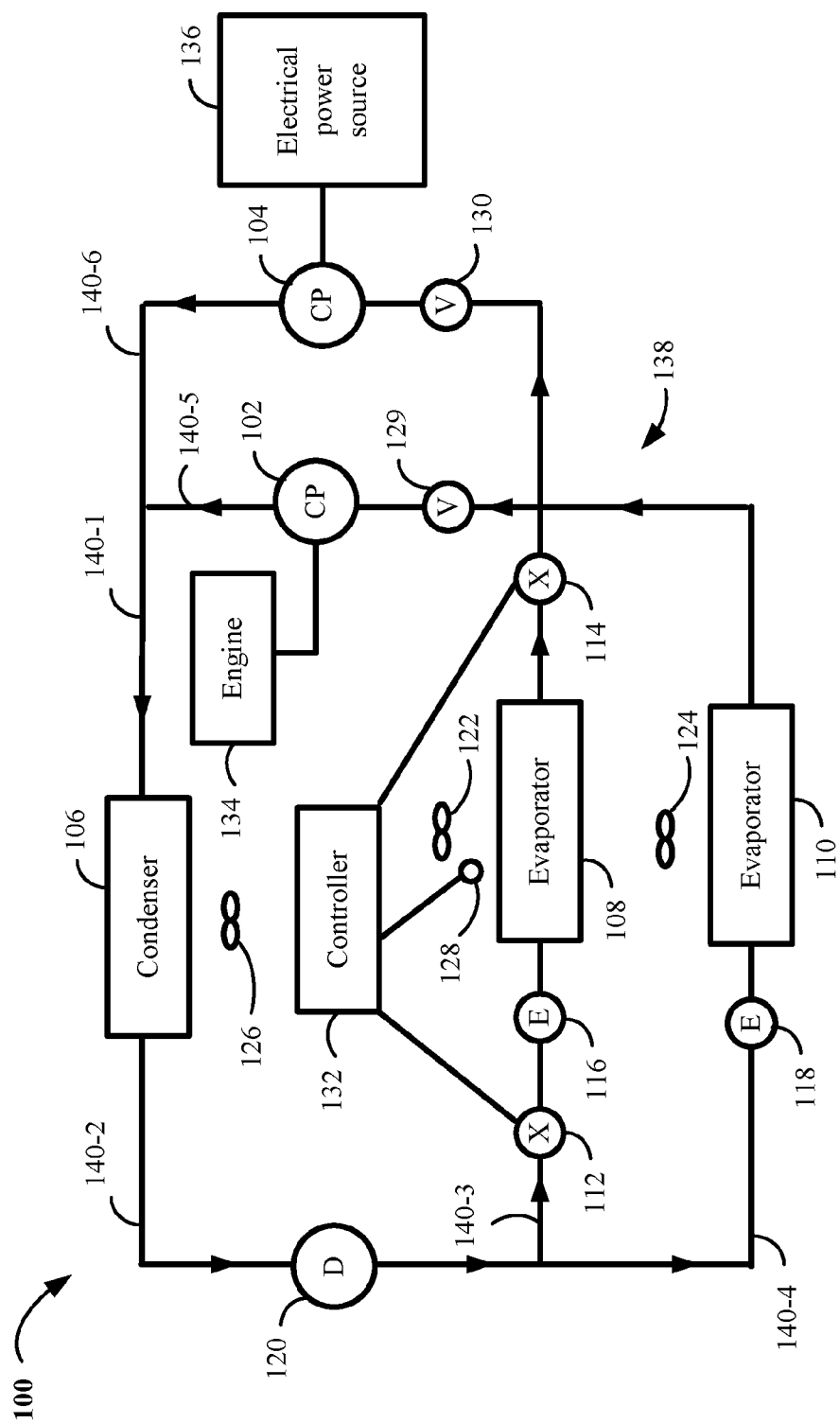


FIG. 1

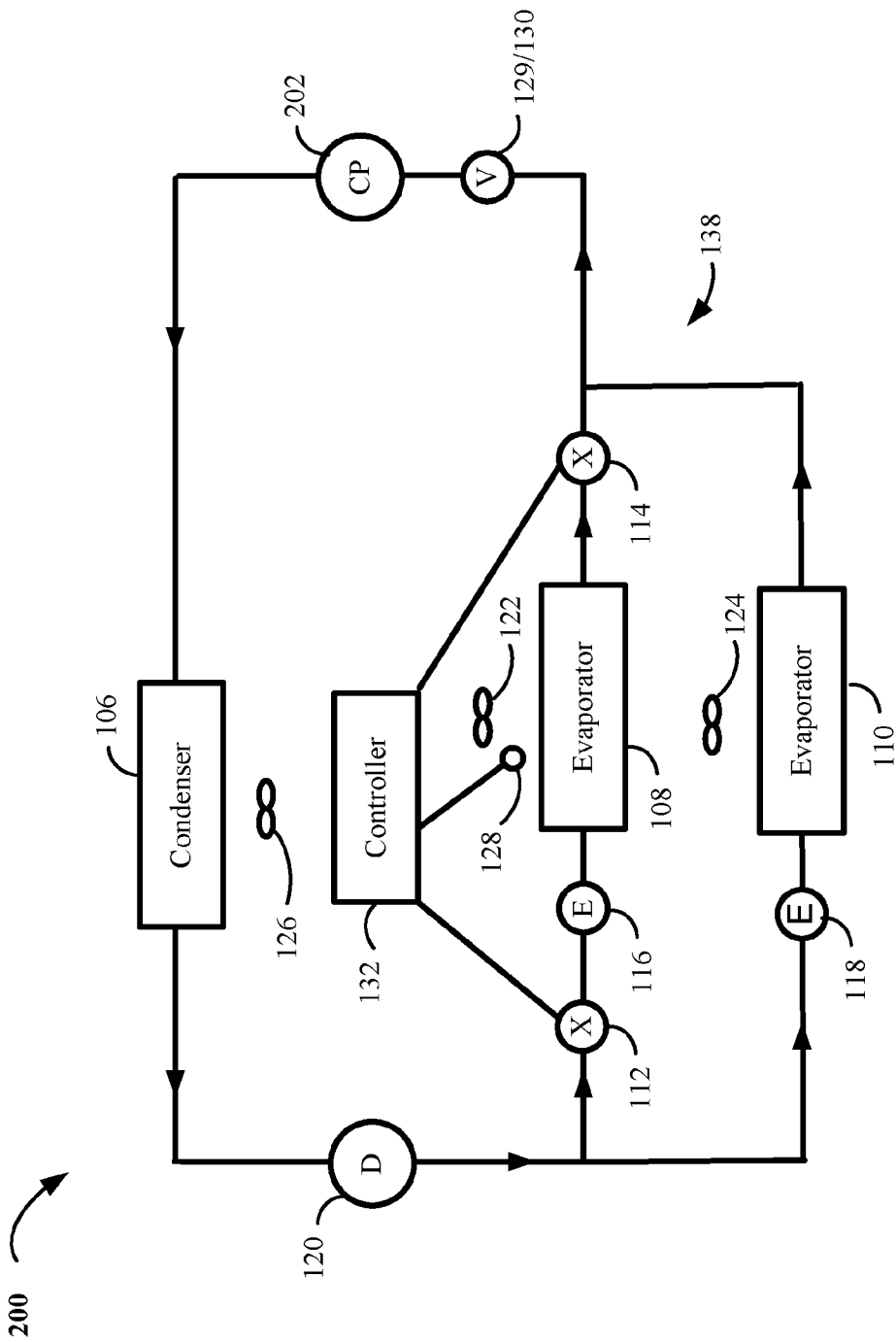


FIG. 2

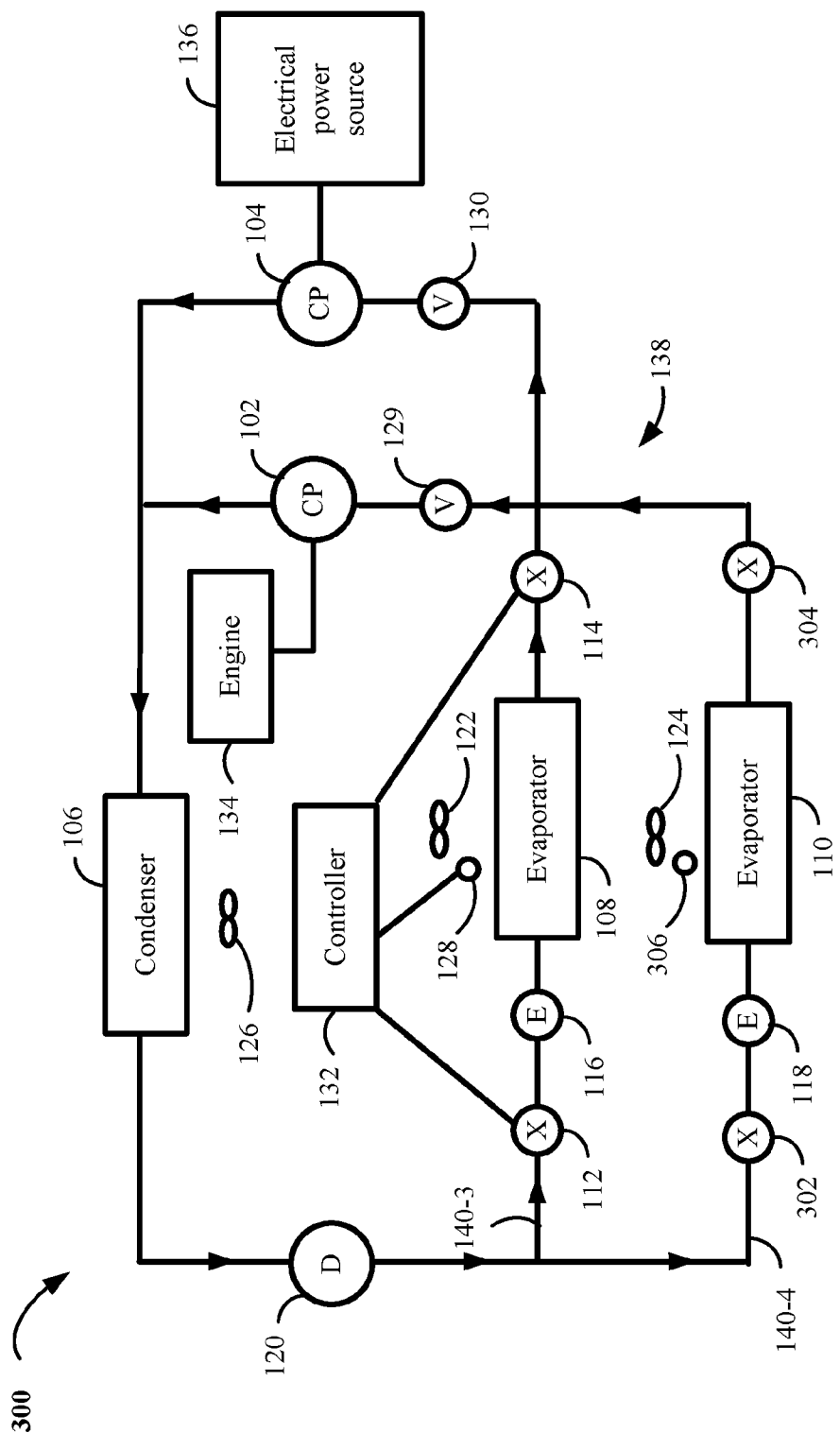


FIG. 3

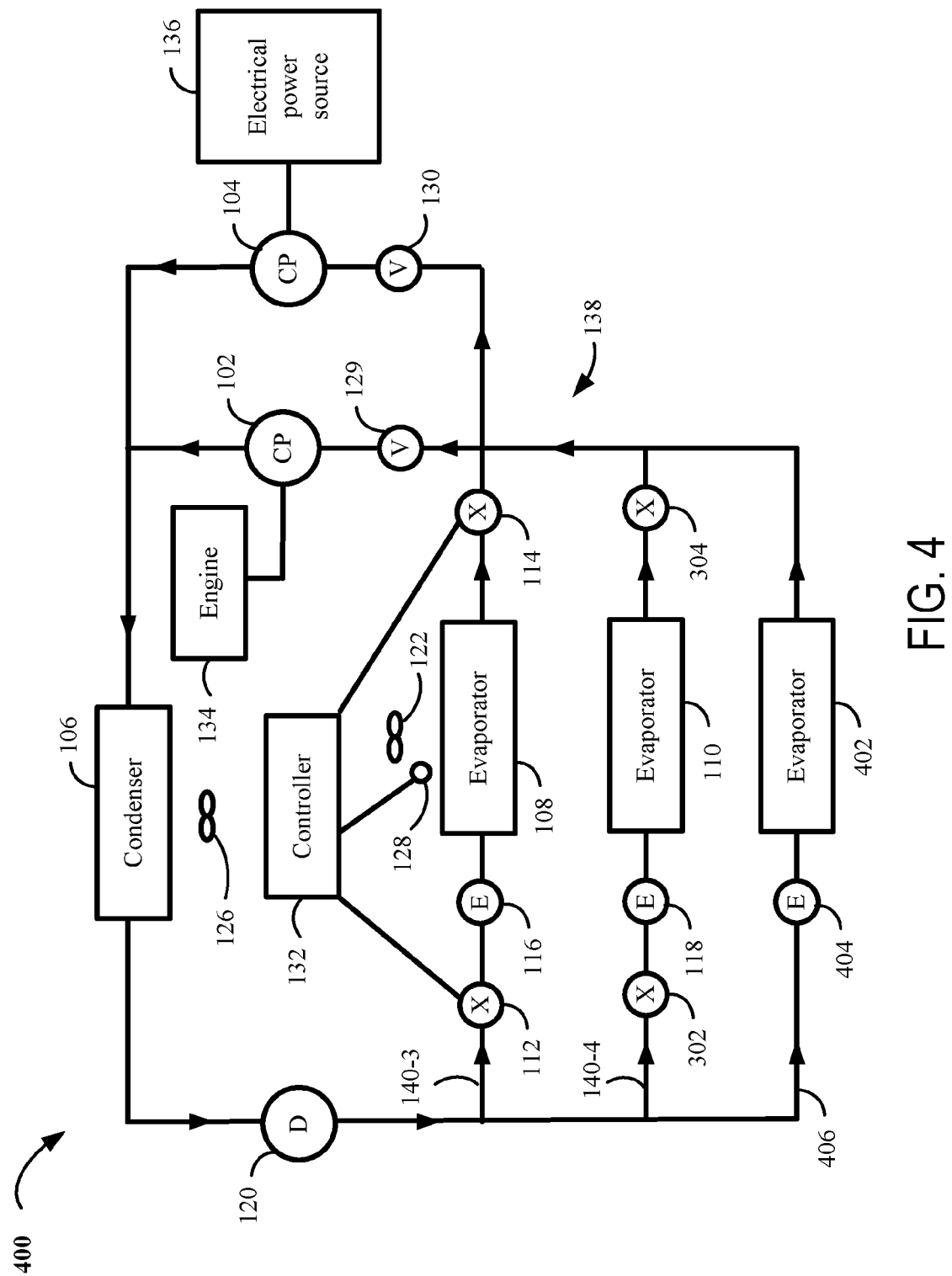


FIG. 4

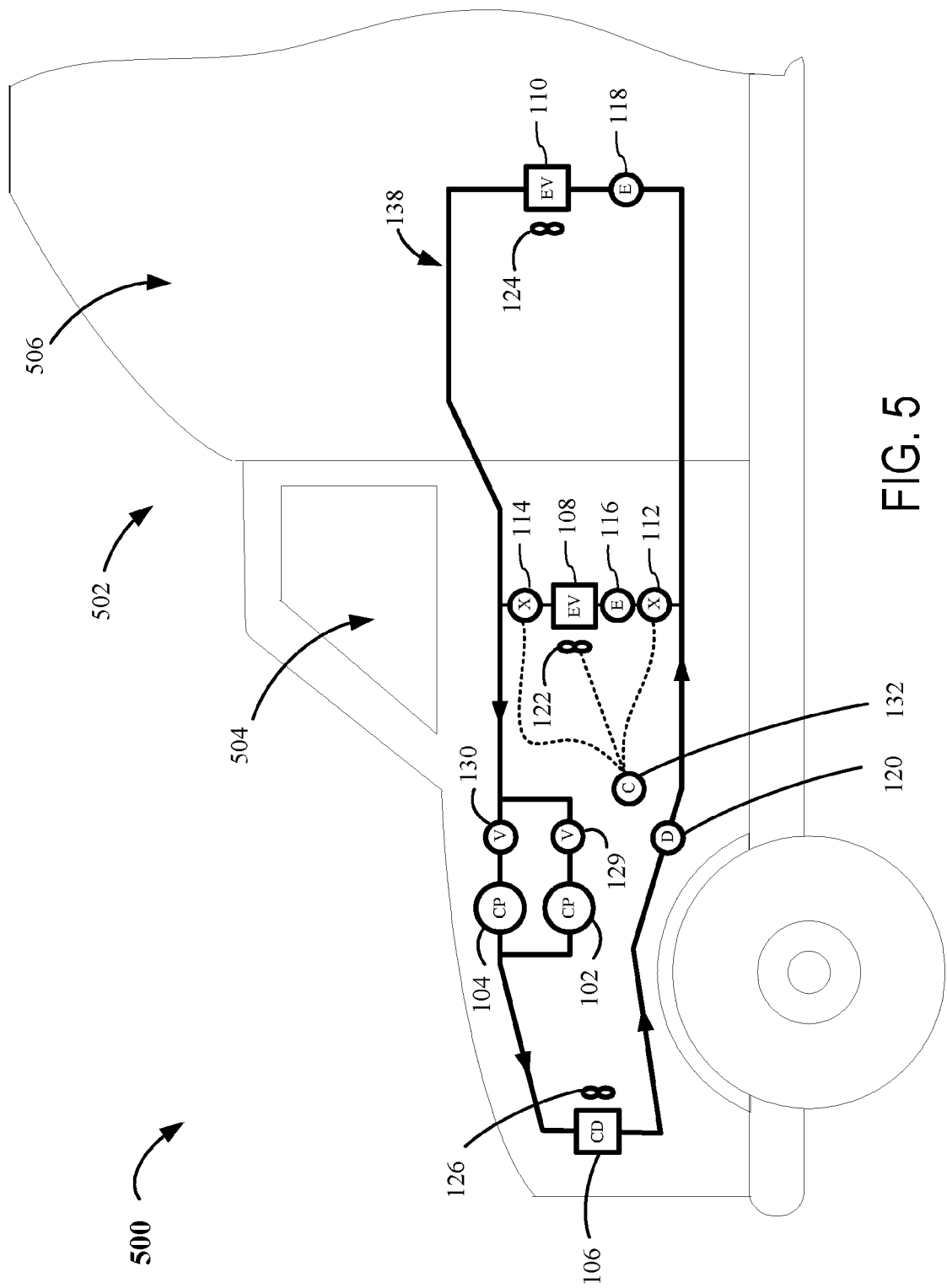


FIG. 5

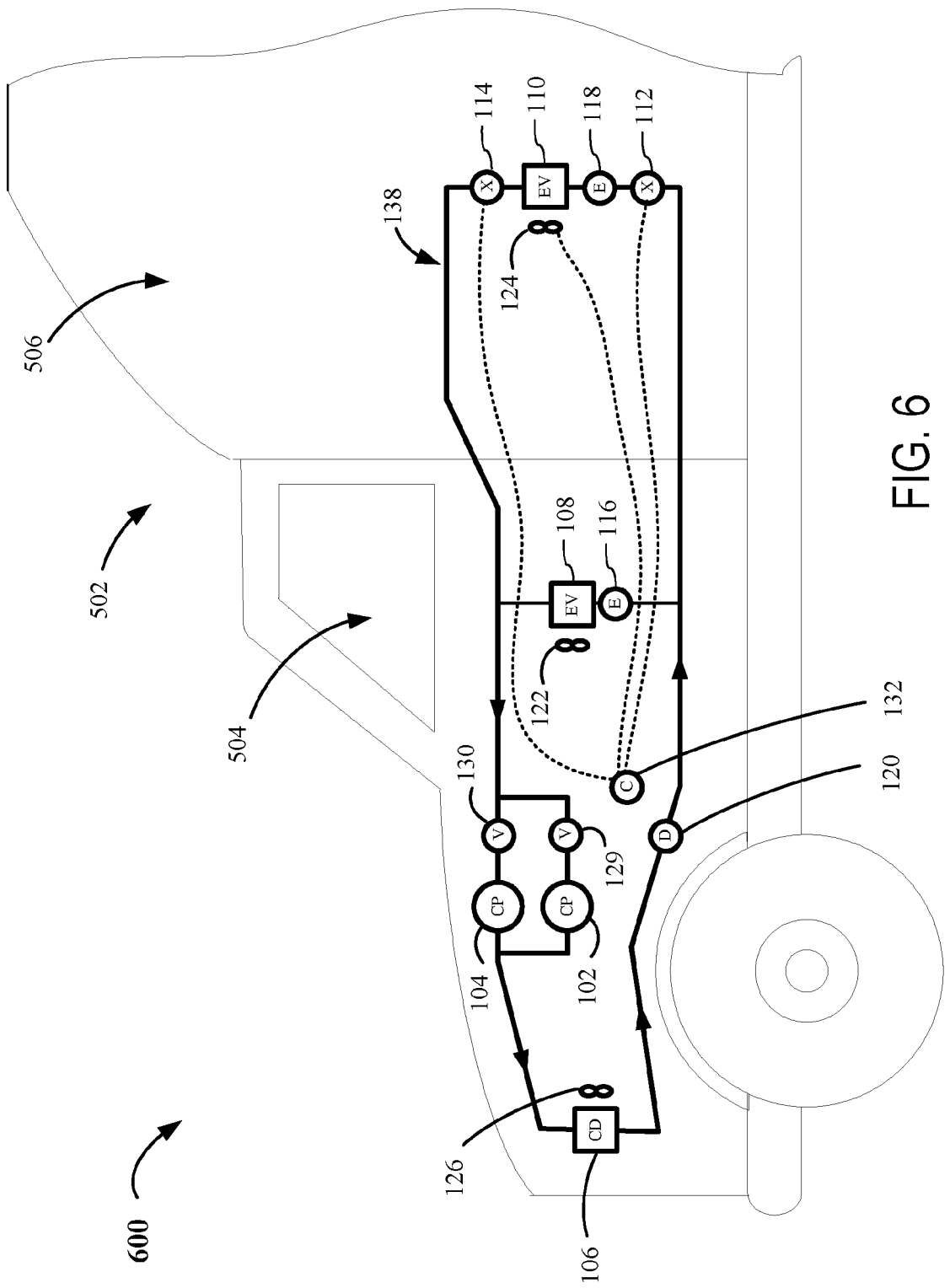
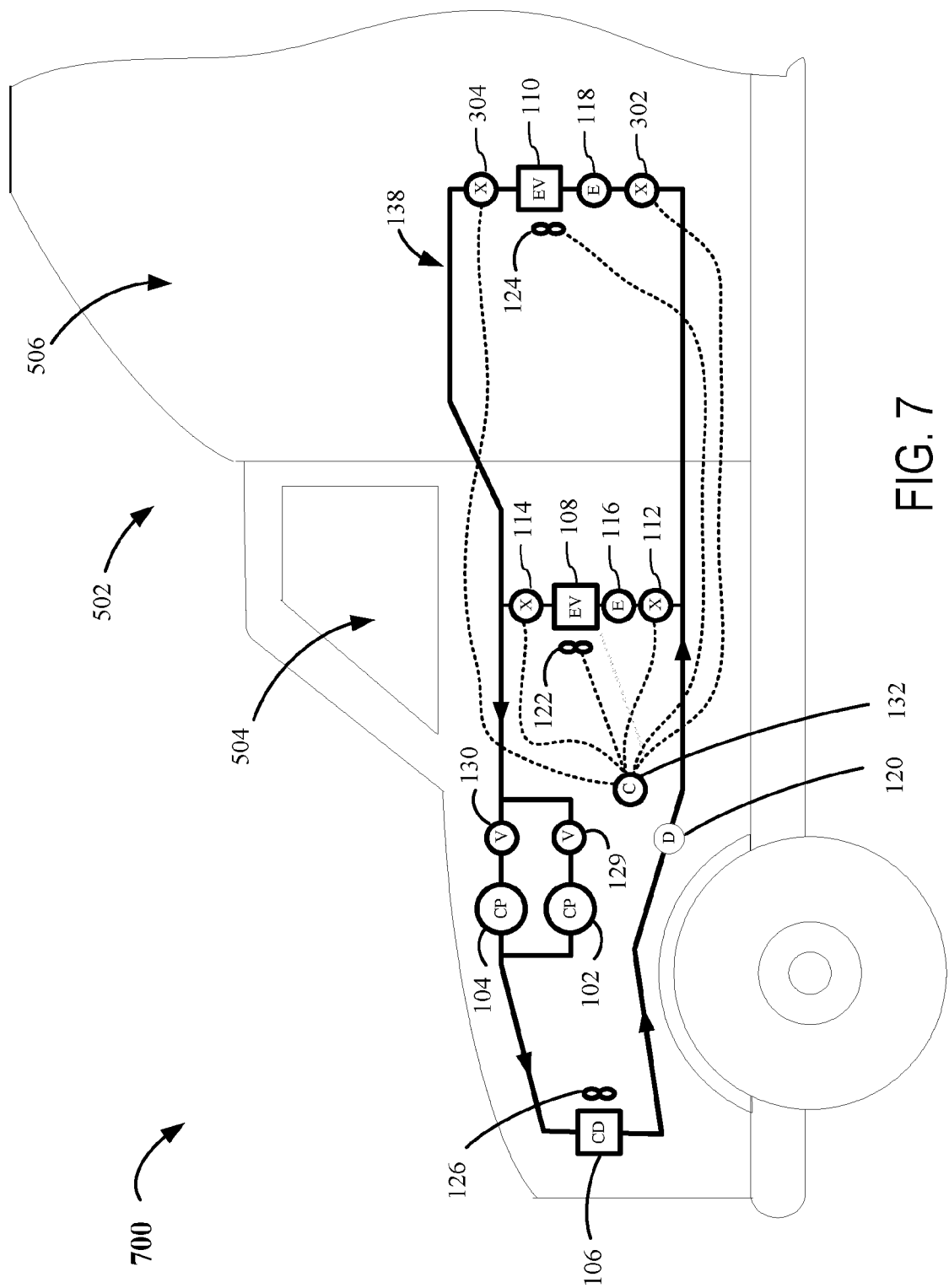


FIG. 6



INTERNATIONAL SEARCH REPORT

International application No
PCT/US2016/065812

A. CLASSIFICATION OF SUBJECT MATTER
INV. B60H1/32 B60H1/00
ADD.

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

B60H F25B F25D

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)

EP0-Internal, WPI Data

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	US 6 543 245 B1 (WALDSCHMIDT JOHN ELLIOTT [US] ET AL) 8 April 2003 (2003-04-08)	1-3,11, 13-19
Y	column 2, lines 10-24; column 4, lines 33-39;; claim 1; figures 2,3	6-10,20, 21

X	US 2009/211280 A1 (ALSTON GERALD ALLEN [US]) 27 August 2009 (2009-08-27)	1,4,5, 11-13, 15,18
A	paragraphs [0034] - [0035;0066]; figure 4	2,3

Y	WO 2014/209780 A1 (CARRIER CORP [US]) 31 December 2014 (2014-12-31)	6
A	paragraphs [0018], [0023], [0026] - [0028]; figure 2	2,3

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☒ Further documents are listed in the continuation of Box C.

☒ See patent family annex.

* Special categories of cited documents :

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"O" document referring to an oral disclosure, use, exhibition or other means

"P" document published prior to the international filing date but later than the priority date claimed

"T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention

"X" document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone

"Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art

"&" document member of the same patent family

Date of the actual completion of the international search

8 March 2017

Date of mailing of the international search report

22/03/2017

Name and mailing address of the ISA/

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INTERNATIONAL SEARCH REPORT

International application No
PCT/US2016/065812

C(Continuation). DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
Y	US 2007/131408 A1 (ZEIGLER TERRY [US] ET AL) 14 June 2007 (2007-06-14) cited in the application	7-10,20, 21
A	paragraphs [0059] - [0060]; figure 7 -----	1,5
X	JP H02 128915 A (NIPPON DENSO CO) 17 May 1990 (1990-05-17) abstract; figure 1 -----	1,5,11, 13,15,18

INTERNATIONAL SEARCH REPORT

Information on patent family members

International application No

PCT/US2016/065812

Patent document cited in search report	Publication date	Patent family member(s)	Publication date
US 6543245	B1	08-04-2003	DE 10252072 A1 18-06-2003
			JP 2003148851 A 21-05-2003
			US 6543245 B1 08-04-2003

US 2009211280	A1	27-08-2009	NONE

WO 2014209780	A1	31-12-2014	EP 3014197 A1 04-05-2016
			US 2016138836 A1 19-05-2016
			WO 2014209780 A1 31-12-2014

US 2007131408	A1	14-06-2007	AT 490878 T 15-12-2010
			AU 2008200554 A1 11-09-2008
			BR PI0800271 A 07-10-2008
			CA 2621931 A1 22-08-2008
			CN 101251096 A 27-08-2008
			EP 1985481 A2 29-10-2008
			JP 2008201411 A 04-09-2008
			US 2007131408 A1 14-06-2007

JP H02128915	A	17-05-1990	NONE
