

## [54] METAL WORKING APPARATUS

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[51] Int. Cl. .... **B21c 23/00**

[58] Field of Search .... **72/325, 256, 266, 267**

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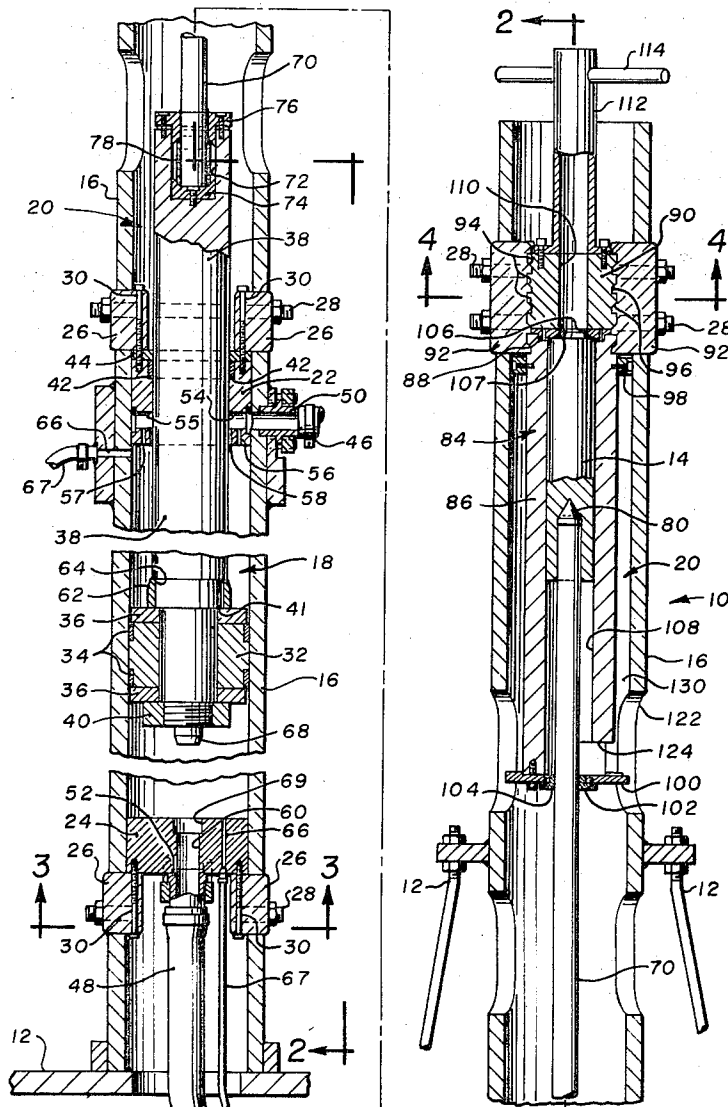
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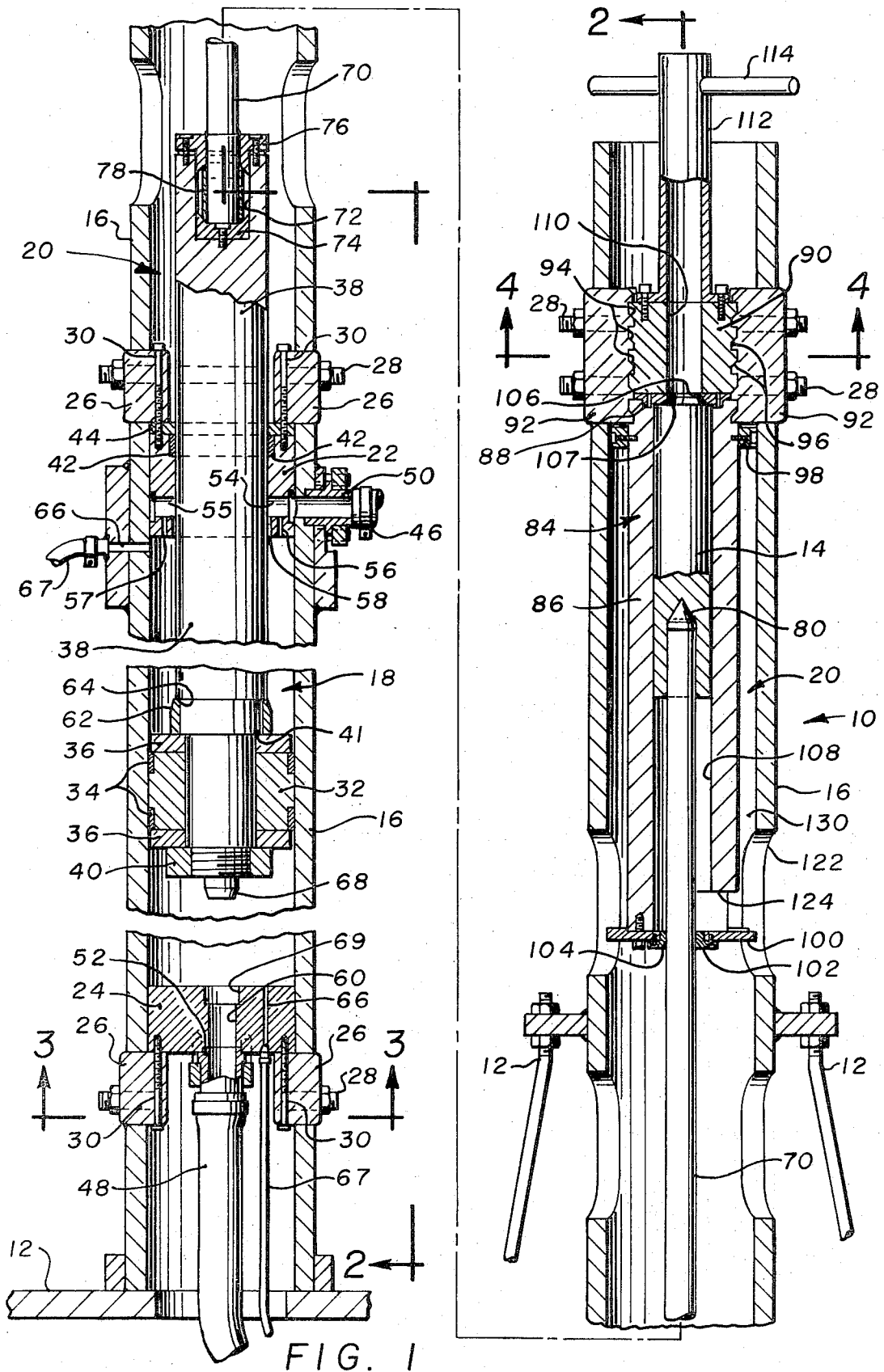
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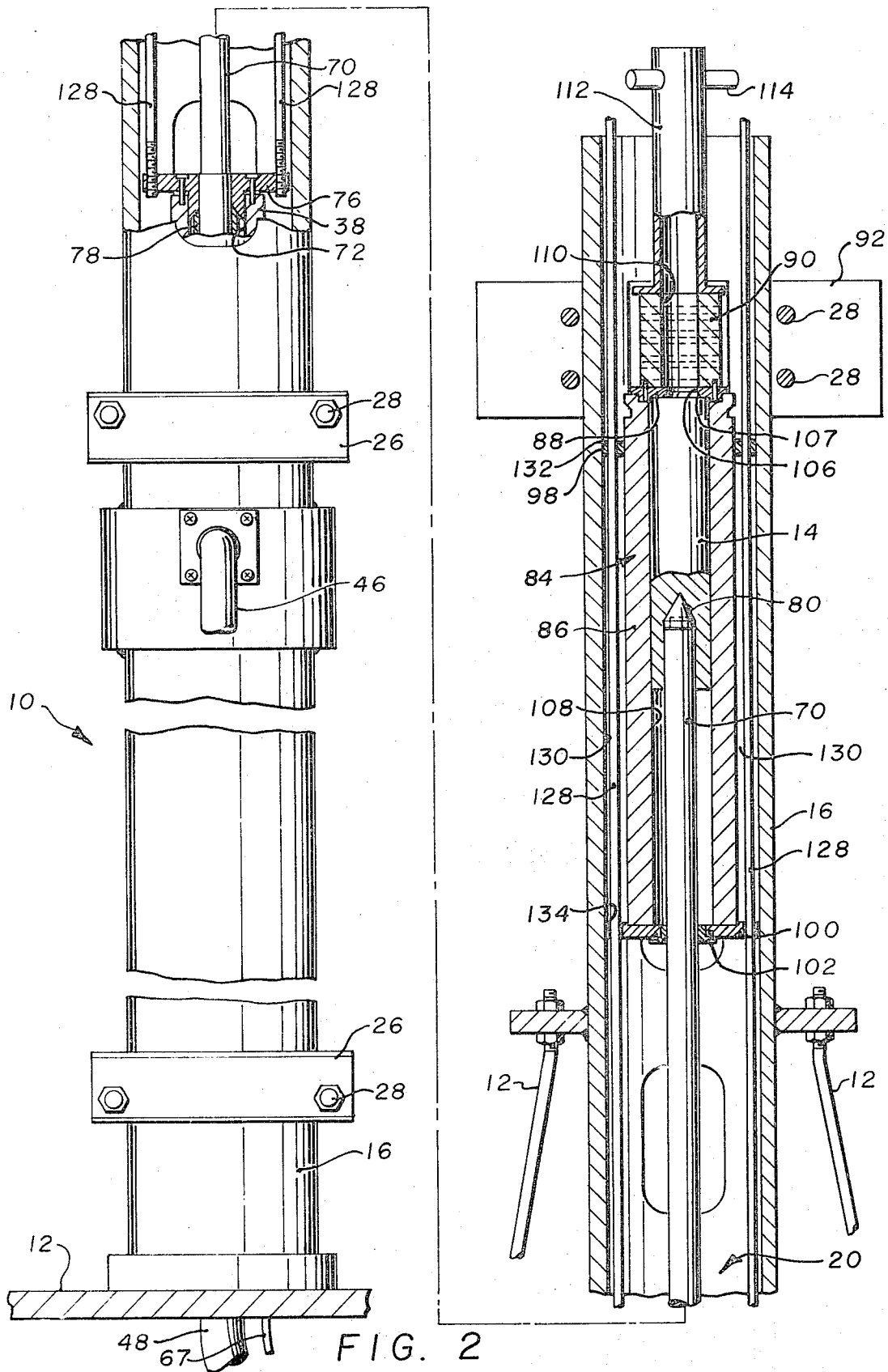
**ABSTRACT**

A press for metal working of cylindrically shaped metal ingots or billets. The press comprises a longitudinally extending hollow tubular member in which a pair of cylinder end members are supported. The two cylinder end members divide the interior of the tubular member into a cylinder portion therebetween and a working portion. A mandrel, aligned with the axis of the tubular member, is provided in the working portion of the tubular member and is connected to a piston provided in the cylinder portion. A die means having a billet chamber and a longitudinal passage there-through to permit passage of the mandrel into and through the billet chamber is supported in the working portion of the tubular member so as to have the axis of the billet chamber and the longitudinal passage aligned with the axis of the tubular member. Supply means are provided for supplying fluid to the cylinder portion of the tubular member to drive the piston longitudinally between the two cylinder end members and accordingly the mandrel to perform the metal working on the billet.

**8 Claims, 9 Drawing Figures**







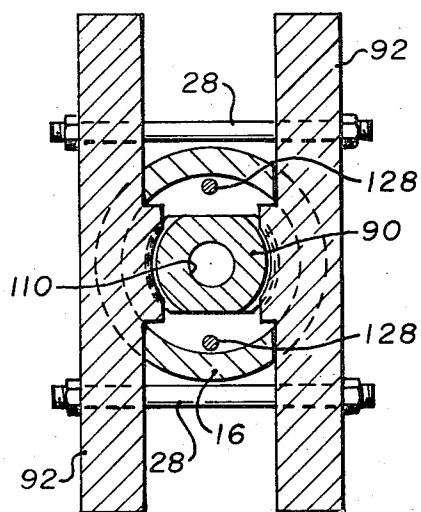


FIG. 4

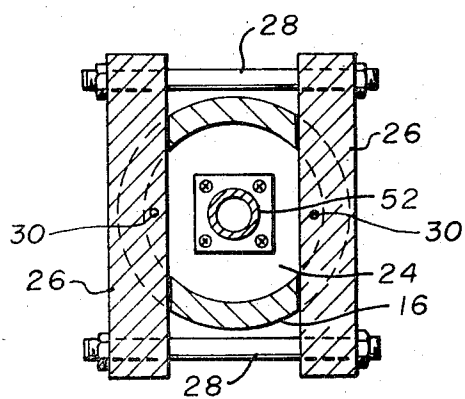


FIG. 3

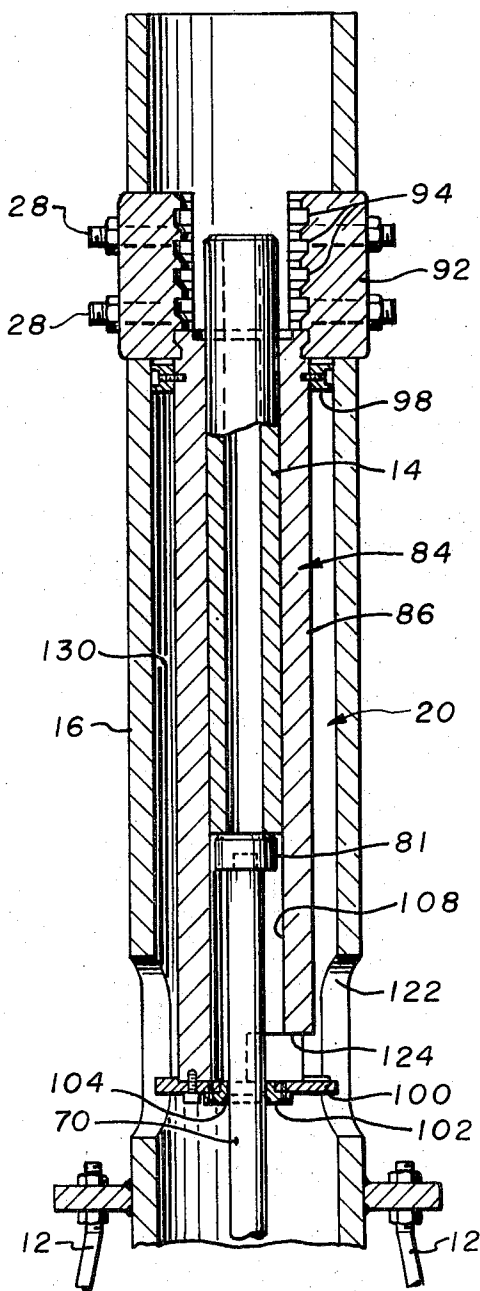


FIG. 5

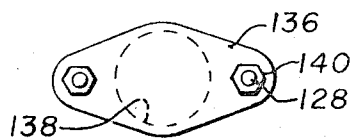


FIG. 7

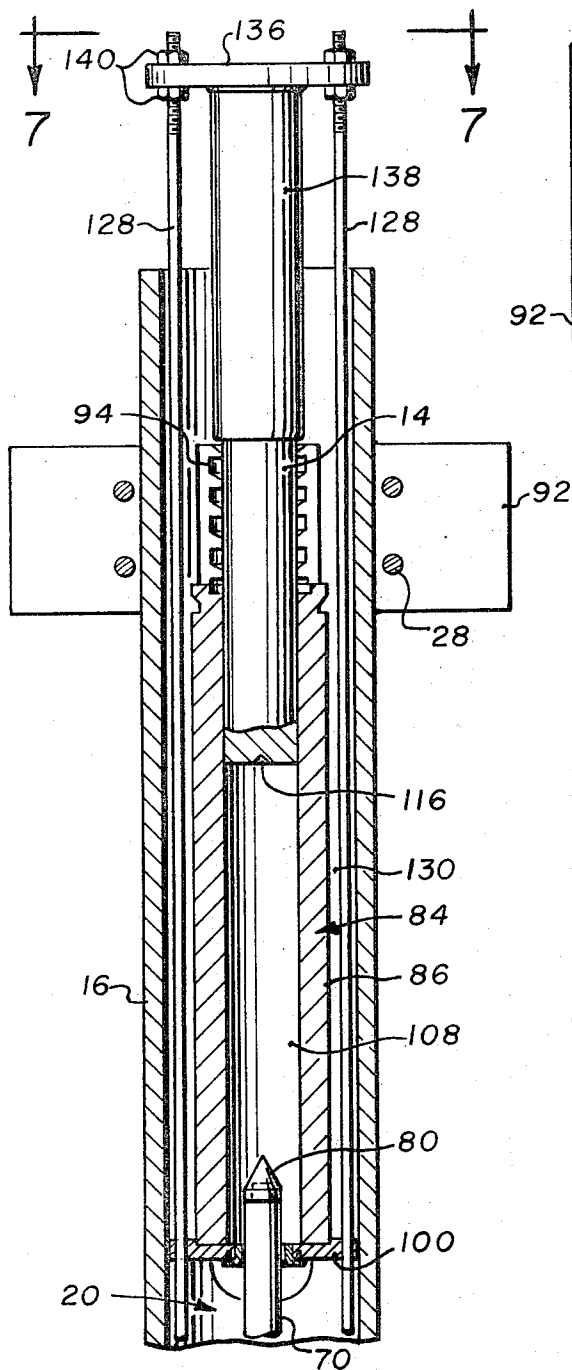


FIG. 6

FIG. 9

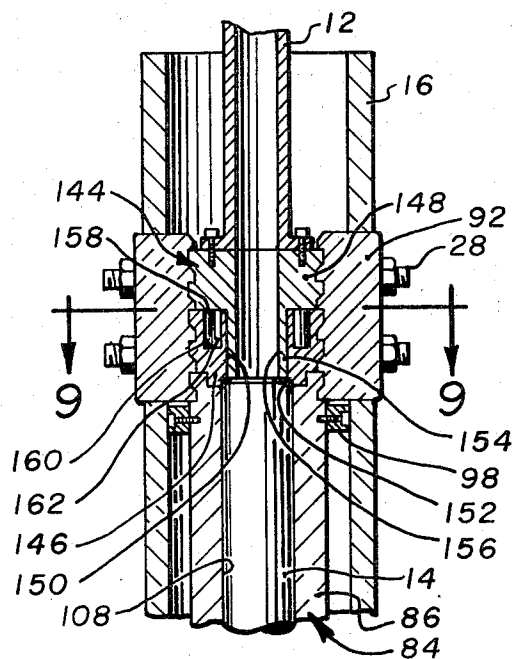
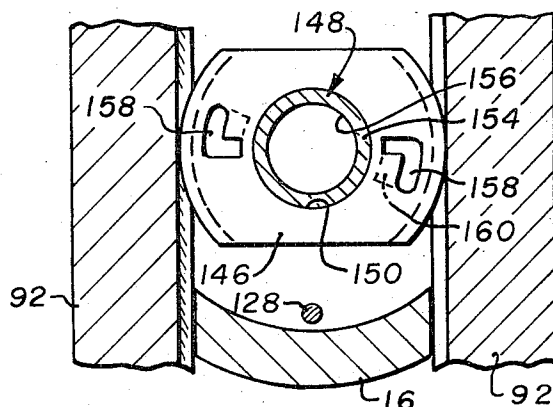


FIG. 8

## METAL WORKING APPARATUS

## BACKGROUND OF THE INVENTION

The present invention relates to the art of metal working and more particularly to an apparatus for metal working of metal ingots. Further still the invention concerns an apparatus for piercing and/or extruding a cylindrical metal ingot.

In one method of making a high quality alloy seamless tube, the first step is to pierce a hole in a relatively short, thick solid circular cylinder called a billet. Normally this is done after heating the billet to 2,000°F or more. During this piercing process in which a mandrel is forced through the heated billet, the billet will lengthen to form what is known as a tube hollow or simply a hollow. As can be appreciated, one of the critical factors in piercing is to maintain the concentricity of the pierced hole throughout the billet length. This hot working of the billet is necessary to give the billet sufficient refinement to permit the tube hollow to be cold worked to form the final seamless tube. In some instances, depending on the billet constituents and the final tube geometry, it is necessary to further hot work the pierced billet. In such situations, the tube hollow can be extruded after piercing in a second operation. Extrusion consists of forcing the heated hollow through a restriction while controlling the internal hole size by a mandrel, thereby lengthening the hollow.

Conventional mechanical and hydraulic presses for piercing and extruding usually have two spaced platens which move toward one another during operation. One of the platens has a billet container positioned thereon in alignment with a mandrel fixed to the other platen. A billet is placed within the container between the two platens and the two platens are moved toward one another to perform the metal working of the billet. As can be appreciated, in order to obtain a high quality pierced and/or extruded billet, the mandrel must be coaxial with the billet container and the mandrel must also move in a straight line toward the center of the container. Elaborate devices have been designed for aligning and guiding the mandrel and billet container during this operation. However, these devices are very expensive and have not always proven to be successful in producing concentrically pierced and extruded billets.

Furthermore, the loading of the billet between the two platens in a conventional press leads to other problems. First, assuming that the mandrel is attached to the upper platen as is normally the case, the upper platen must be withdrawn high enough that the mandrel does not obstruct loading of the billet into the billet container. This longer stroke requirement increases equipment cost and may detract from concentricity. Secondly, the upper platen limits the mechanical means which are available for lifting of a hot billet into the container.

## SUMMARY OF THE INVENTION

Accordingly, there is provided a novel apparatus for metal working of cylindrically shaped ingots. The apparatus comprises a longitudinally extending hollow tubular member in which a pair of axially spaced cylinder end members are transversely supported. The two cylinder end members divide the interior of the tubular member into a cylinder portion located between the

two end members and a working portion. A piston is provided in the cylinder portion for longitudinal movement between the two end members. A longitudinally extending working mandrel, having at least a portion of its longitudinal length disposed in the working portion, is aligned with the longitudinal axis of the tubular member and is connected to the piston for movement therewith. A die means, having a longitudinally extending billet chamber and a longitudinal passage therethrough to permit passage of the working mandrel into and through the billet chamber, is supported in the working portion of the tubular member so that its axis is aligned with the axis of a tubular member. Supply means is provided for supplying fluid to the cylinder portion to move the piston longitudinally therewithin and accordingly to move the mandrel to perform the metal working on the cylindrically shaped ingot.

The use of a single multi-purpose tubular member which serves the multiple purposes of being the main body of the press, the hydraulic cylinder which motivates the operation, the means of retaining and accurately aligning the billet container with the piston rod and the mandrel, and the structural member which contains large forces and reactions between the members is largely responsible for achieving an economical metal working apparatus and for achieving excellent alignment of the components. This is an essential condition for producing concentric holes in metal billets. Furthermore, the axis of the billet is vertical rather than horizontal. This keeps the weight of the billet, the container, the piston rod, and the piercing mandrel from contributing directly to the eccentricity as might otherwise occur with a horizontal press. It also lessens the indirect effects such as the billets' tendency to lay in one side of the container and be cooled asymmetrically. Further still the press is axisymmetrical in design which minimizes the asymmetrical temperatures and asymmetrical strains as possible causes of eccentricity. Still further the press permits unobstructed billet leading of the container at the opposite end from the mandrel.

## BRIEF DESCRIPTION OF THE DRAWINGS.

FIG. 1 is an elevation view in section of a metal working press according to the present invention.

FIG. 2 is a side elevation view partially in section taken along line 2—2 of FIG. 1.

FIG. 3 is a cross sectional view taken along line 3—3 of FIG. 1.

FIG. 4 is a cross sectional view taken along line 4—4 of FIG. 1.

FIG. 5 is an elevation view similar to the upper portion of FIG. 1 showing how a pierced billet may be extracted from the metal working press.

FIG. 6 is a side elevation view similar to the upper portion of FIG. 2 showing how a billet may be forcibly inserted into the metal working press.

FIG. 7 is a plan view taken along line 7—7 of FIG. 6.

FIG. 8 is an elevation view in section showing an alternative arrangement for the billet chamber end member of the press.

FIG. 9 is a sectional view taken along line 9—9 of FIG. 8.

## DESCRIPTION OF PREFERRED EMBODIMENT

Referring now to the drawings, FIGS. 1 and 2 show

a metal working press 10 for metal working of a short, thick generally circular cylinder 14 called a billet. The billet 14 is normally heated to about 2000°F or more in order that it may be hot worked thereby giving it more ductility. The increased ductility will allow the hot worked billet to then be cold worked into high quality seamless tubing. In particular, the press 10 is a piercing and/or extrusion press although it may easily be modified to perform other types of metal working operations. In the preferred embodiment the press 10 is vertically mounted and supported on and by a base 12.

The press 10 is comprised of an elongated multi-purpose hollow tubular member 16. Upper and lower cylinder end members 22 and 24 are supported within the interior of the tubular member 16 to divide the interior into a cylinder portion 18 defined between the two members 22 and 24 and a working portion 20 defined by the remainder of the interior of the tubular member 16. The cylinder portion 18 forms the hydraulic cylinder of the press 10 which motivates the metal working operation. Each of the two end members 22 and 24 are supported by means of two diametrically opposed, bolt stabilized transverse keys or bars 26. The bars 26 are disposed in diametrically opposed openings in the side wall of the tubular member 16 adjacent the desired elevations of the end members 22, 24. The openings are each of a rectangular configuration in order that a portion of each bar 26 extends into the working portion 20 of the tubular member 16 to overlay a portion of one of the cylinder end members 22 or 24. The interior lying portion of each of the bars 26 is bolted by bolts 30 to the cylinder end member 22 and 24. Outside of the tubular member 16, the two bars 26 adjacent each elevation of the cylinder end members are bolted to each other by bolts 28 to stabilize and retain the bars 26 within the openings in the side wall of the tubular member 16. This provides an economical means for positioning the two end closures 22, 24 of the hydraulic cylinder 18 in the tubular member 16 and for transferring the load imposed thereon to the tubular member 16 without interrupting its basic continuity.

A piston 32 whose outside diameter is substantially equal to the inside diameter of the tubular member 16 is provided in the cylinder portion 18 for longitudinal movement between the two cylinder end members 22 and 24. Piston sealing means 34, such as for example leather chevron packing seals, are positioned in recesses in the outer cylindrical surface of piston 32 and are held there in place by seal hold rings 36. One end of a piston rod 38 passes through a central bore in the piston 32 and seal hold rings 36 and is connected thereto by means of a nut 40 which forces the piston 32 and seal hold rings 36 against the shoulder 41 on piston rod 38. This arrangement further serves to maintain a proper relationship between the piston 32 and seal hold rings 36. The other end of the piston rod 38 extends longitudinally upward through a central bore in the upper end member 22 into the working portion 20 of the tubular member 16. Upper cylinder end member seals 42, which again may be leather chevron packing seals, are provided in the bore of the upper end member 22 to seal the cylinder portion 18 from the working portion 20. A seal retaining ring 44 is provided and retained in place between the upper end member 22 and the interior lying portion of the transverse bars 26 dry bolts 30.

If desired, a pliable washer (not shown) may be provided between the retaining ring 42 and the bars 26 for keeping debris free from the piston rod 38 as it moves longitudinally through the cylinder end member 22.

Fluid lines 46 and 48 are coupled or clamped respectively to fluid connectors 50 and 52 for supplying actuating fluid to the cylinder 18 to drive the piston 32 longitudinally between end members 22 and 24. Actuating fluid is supplied, in a well-known manner, from a fluid source and control mechanism (not shown) to which the other end of lines 46 and 48 are connected. Upper fluid connector 50 is bolted to the sidewall of the tubular member 16 and communicates with side port 54 in the upper end member 22. Side port 54 in turn communicates with cylinder portion 18 above the piston 32 through longitudinal port 56 and annular opening 58. Similar side and longitudinal ports 55, 57 are provided in the end member 22 diametrically opposite to the ports 54, 56 are for insuring a minimum fluid communication to the cylinder portion 18 when the annular opening 58 is closed as described hereinbelow. Lower fluid connector 52 is bolted to the lower cylinder end member 24 and communicates with the cylinder portion 18 below piston 32 through central bore 60 in the lower end member 24. Pressure relief opening 66 to which fluid lines 67 are coupled are provided near the upper and lower end members 22 and 24 in order to limit and control the buildup of fluid pressure in the cylinder portion 18 when the piston moves toward either one of the upper or lower end members. A tapered ring 62 is attached to the piston rod 38 between a shoulder 64 on the piston rod 38 and the upper seal hold ring 36. The tapered ring 62 acts as a cushion or decelerator to slow the upward movement of the piston 32 by gradually closing off the fluid passage through annular opening 58 as the piston is moved upward. The lower tapered end 68 of the piston rod 38 provides a similar cushioning or decelerating effect as it enters the enlarged portion 69 of the central bore 60 in the lower end member 24 when the piston 32 is moved downward.

A longitudinal extending working mandrel 70, aligned with the longitudinal axis of the tubular member 16, is attached to the upper end of the piston rod 38. In the preferred embodiment a collet chuck is used for this purpose which comprises a base member 74 fixed in a central bore 72 in the piston rod 38 and having an upper annular tapered portion, an upper plate member 76 having a lower annular tapered portion, and a segmented ring member 78 having complementary upper and lower tapered portions engageable with the tapered portions of the base member 74 and the plate member 76. The plate member 76, through which the lower end of the mandrel 70 extends into the central bore of the piston rod 38 is bolted to the piston rod 38 to bias the segmented ring 78 inwardly to center and frictionally hold the lower end of the mandrel 70. It should be noted that the use of a collet chuck is not the only means for attaching and aligning the lower end of the mandrel 70 with the piston rod 38. Other alternatives may include threading or welding of mandrel 70 to the end of piston rod 38. The other end of the mandrel 70 is provided with a removable working tip 80 for performing the desired working operation on the billet 14. In FIGS. 1 and 2 this working tip is a piercing tip 80. The piercing tip 80 may be removed and replaced, for example, with an extraction plug 81 (see FIG. 5) or

an extrusion tip depending upon the desired operation for the press 10.

An upper die means 84 for the billet 14 is positioned in the upper end of the working portion 20 of the tubular member 16. In the embodiment shown in FIGS. 1 and 2 the upper die means 84 comprises a billet container 86 having a billet chamber 108, a billet centering plate 88, and a chamber end member 90. These three components 86, 88 and 90 are supported within the tubular member 16 by means of a breech lock arrangement which in turn is supported by the tubular member 16 in a manner similar to the support of the cylinder end member transverse keys 26. Breech lock keys or bars 92, each of which has an inner lateral portion having a series of convolutions 94 formed on the longitudinal surface thereof, are disposed in diametrically opposed openings in the sidewall of the tubular member 16 such that the inner lateral portions lie within the working portion 20 of the tubular member 16 and are diametrically opposed to one another. The bars 92 are retained thereon by cross bolts 28 which pass outwardly of the tubular member 16. Complimentary convolutions 96 are provided on opposite portions of the longitudinal surface of the die means 84, in particular on opposite portions of the billet container 86 and the chamber end member 90, for engagement with the convolutions 94 of the breech lock keys 92 to support and retain the die means 84 in the working portion 20 of the tubular member 16. As can be seen in FIG. 1, the convolutions 94, 96 are preferably tapered to make them stronger at the base of the longitudinal surfaces where the shear forces are the greatest.

The longitudinal extending billet container 86 has a central bore therethrough which defines the billet chamber 108 in which the billet 14 is placed. Upper and lower centering rings 98, 100 are attached to the outer surface of the billet container 86 adjacent the upper and lower ends respectively thereof to align the billet chamber 108 with the longitudinal axis of a tubular member 16. The lower centering ring 100, which is bolted to the bottom of the container 86, has a lower mandrel centering plate 102 bolted thereto for centering and aligning the mandrel 70 as it passes therethrough into the billet chamber 108. The tapered entrance 104 of the centering plate 102 serves to guide the mandrel 70 into this alignment at the lower end of the billet chamber 108.

The chamber end member 90 forms the upper end closure for the billet chamber 108 and sustains the reaction of the billet as it is worked by the mandrel 70. The billet centering plate 88 is bolted to the bottom of the chamber end member 90 so as to have its central opening 106 aligned with the billet chamber 108 when the chamber end member 90 is locked in place by the breech lock keys 92. The billet centering member 88 is provided with a tapered entrance 107 to the opening 106 which serves to initially align the billet 14 in the billet chamber 108, as described herein below. As can be seen in FIGS. 1 and 2, the chamber end member 90 has a central bore 10 therethrough in alignment with the central opening 106 in the billet centering member 88 to allow for passage of the mandrel 70 therethrough on its upward stroke during a piercing metal working operation. The upper end of the chamber end member 90 has a tubular extension member 112 bolted thereto to which a handle 114 is attached for rotating the chamber end member 90 relative to the breech lock

keys 92 in order to facilitate removal from the breech lock arrangement. As can be seen in FIG. 4, the necessary rotation for removal of a chamber end member 90 (and accordingly the billet centering plate 88) is approximately 90°.

The operation of the press 10 for a piercing operation is as follows. The chamber end member 90 with attached billet centering plate 88 is removed from the breech lock arrangement and a heated billet 14 is inserted into the chamber 108. The billet preferably has a centralized recess 116 in the lower surface thereof (see for example FIG. 6 although this FIG. shows a billet 14 being forcibly inserted into the chamber 108) which is aligned with the billet axis and into which will fit the piercing tip 80 of the mandrel 70. This will center the lower end of the billet 14 in the billet chamber 108. Also the billet 14 is preferably of such a size that it loosely fits into the billet chamber 108 so that on the first upward action of the mandrel 70, it will force the billet into the tapered entrance 107 of the billet centering plate 88 to center the upper end of the billet 14 in the chamber 108. This use of centralized recess 116 and tapering entrance 107 improves the symmetry and concentricity of the resulting pierced billet by insuring that the billet 14 is aligned at both its upper and lower ends. After the billet 14 has been inserted in the chamber 108 and the chamber end member 90 locked in place, fluid is introduced through line 48 into the cylinder 18 below the piston 32 in order to drive the piston upward. This causes the mandrel 70 to pass through the lower centering plate 102 into the lower end of billet chamber 108 where piercing tip 80 of the mandrel 70 engages the centralized recess 116 of the billet 14 and forces the billet 14 upward and into alignment in the billet container 108 as described hereinbefore. The mandrel 70 continues upward, thereby piercing the billet 14, until the tapered ring 67 on the piston rod 38 engages the annular opening 58 in the cylinder upper end member 22 to terminate the upward movement of the piston 32. During this piercing operation, the billet 14 will grow longitudinally and accordingly the lower end of the pierced billet 14 will move downwardly (since it is retained at the top by the chamber end member 90) in the billet chamber 108. Additionally, a small portion of the metal billet 14 will be forced out through the central opening in the billet centering plate 88 and chamber end member 90 as the mandrel 70 passes therethrough to its fully extended position. After the piercing operation has been completed, the piston 32 is retracted within the cylinder 18 of the tubular member 16 by introducing fluid to the cylinder portion 18 through line 46.

In order to remove the pierced billet 14 or tube hollow from the billet chamber 108, the chamber end member 90 is removed and an extraction plug 81 (see FIG. 5) is inserted into the billet chamber 108 adjacent its lower end through opening 122 in the sidewall of member 16 and similar opening 124 in the sidewall of billet container 86. If desired the piercing tip 80 may be removed and extraction plug 81 attached to the end of the mandrel 70, although this would not be necessary if the extraction plug 81 were provided with a centralized recess or opening in which the piercing tip 80 would fit. The extraction plug 81 is sized so that its outer diameter is substantially equal to but slightly less than the diameter of the billet chamber 108 so that it may move longitudinally there within. A second stroke



of the mandrel 70 will then cause the extraction plug 81 to engage the lower end of the pierced billet 14 and force the billet upward out of the billet chamber 108.

As referred to hereinbefore, it is desirable that the heated unpierced billet 14 loosely fit into the billet chamber 108. However, in some instances it may be necessary to forcibly insert the billet 14 into the chamber 108. Furthermore, in other operations it may be desirable to exert a force downwardly instead of upwardly. Accordingly, the press 10 has been provided with two diametrically opposite pull rods 128 which are removably connected to the upper plate member 76 which attaches the mandrel 70 to the piston rod 38. As can best be seen in FIGS. 2, 4, 6 and 7 the pull rods 128 extend longitudinally upward through the annular space 130 between the inner surface of tubular member 16 and the outer surface of tubular member 16 and the outer surface of billet chamber 86 to above the upper end of tubular member 16. Each of the pull rods 128 extends through appropriately located and sized holes 132, 134 on the upper and lower centering rings 98 and 100. Also, the pull rods 128 are positioned on opposite sides of the chamber end member 90 between the breech lock keys 92 so as not to interfere with the operation of the breech lock arrangement. The pull rods 128 move with the mandrel 70 and may extend as far above the press 10 as desired. For example, as shown in FIG. 6 and 7, the pull rods 128 are connected together above tubular member 16 cross connecting member 136 in order to be able to forcibly insert a billet 14 into the billet chamber 108. The cross member 136 is coupled to the pull rods 128 by nuts 140 and is provided with a lower extension 138 which will contact and push a billet 14 into the billet chamber 108 as the mandrel 70 is retracted. As can be appreciated, the extension 138 is necessary since the lateral dimension of the cross connecting member 136 is larger than the inside diameter of the billet container 86. After the billet 14 is inserted, the cross connecting member 136 is removed by loosening the nuts 140 and the chamber end member 90 is inserted and locked between the breech lock keys 92.

FIGS. 8 and 9 show and alternative arrangement 144 for the chamber end member 90 to provide for combination piercing and extrusion in which the billet 14 is first pierced and then extruded by a second stroke of a working manual 70. The alternative chamber end member 144 is of a two-part construction having an outer part 146 and an inner part 148. The outer part 146 acts as an extrusion die and has a central bore 150 therethrough whose diameter is slightly less than the diameter of the billet chamber 108. Also there is provided a tapered entrance 152 at the lower end of the outer part 146 which performs the function of the billet centering plate 88 described hereinabove with reference to FIGS. 1 through 7. The inner part 148 has a lower extension 154 which fits into the bore 150 of the outer part 146 and additionally has a central bore 156 to allow passage therethrough of the working manual 70 during the piercing operation. For the piercing operation, both parts 146 and 148 are locked in place by the breeched lock arrangement as shown in FIG. 8 and the billet 14 is pierced in a manner similar to that as previously described. After the mandrel 70 is retracted, only the inner part 148 is removed such as by rotation of the tubular extension 112 attached thereto. This leaves the

outer part 146 in position. An extrusion tip (not shown) may then be attached to the mandrel 70 and a second stroke made which will force the pierced billet 14 or hollow through the reduced diameter of the bore 150 thereby reducing or extruding the diameter of the pierced billet 14.

In order to remove the outer part 146 from the breech lock keys 92, the upper surface of the outer part 148 is provided with two dog legged shaped longitudinal slots 158 located on diametrically opposite sides of the bore 150. Lateral slots 160 extend from these longitudinal slots 158 in the interior of the outer part 146 to provide an upper shoulder 162. Accordingly, an appropriate shaped tool (not shown) having longitudinally extending legs with lateral projections extending inwardly therefrom may be inserted in the longitudinal slots 158 and rotated to rotate the outer part 146 out of engagement with the breech lock keys 92. The lateral projections would then engage the upper shoulder 162 to permit the outer part 146 to be lifted out of the tubular member 16.

Accordingly, the present invention discloses a novel metal working press 10 in which a single multi-purpose tubular member 16 serves the multiple purposes of being the main body of the press 10, the hydraulic cylinder 18 which motivates the operation, the means of retaining and accurately aligning the billet container 86 with the mandrel 70, and the structural member which contains the large force and reactions between the members. Another advantage is that the press 10 minimizes possible causes of eccentricity and improves the symmetry and concentricity of the pierced hole in the billet 14. First, the axis of the billet 14 is vertical and accordingly prevents the weights of the billet 14, the billet container 86 and the mandrel 70 from directly contributing to possible eccentricity. This fact also lessens the indirect effects of eccentricity such as the billet laying on one side and being cooled assymetrically. Secondly, the press 10 in being axisymmetrical in design minimizes assymetrical temperatures and assymetrical strains. Finally, the billet 14 in being pierced is aligned at both its top and bottom ends. A further advantage is that the billet 14 is loaded into the billet container 86 from above out of the way of the press 10 rather than from beneath or between the press as in conventional arrangements. This affords for a more convenient operation of the press 10.

While these preferred embodiments of the invention have been shown and described, it will be understood that they are merely illustrative and that changes may be made without departing from the scope of the invention as claimed.

What is claimed is:

1. Apparatus for metal working of a billet comprising:
  - a longitudinally extending hollow tubular member;
  - a pair of axially spaced end members transversely supported in said tubular member to divide the interior of said tubular member into a cylinder portion located between said two end members and a working portion;
  - a plurality of openings in the side wall of said tubular member adjacent each of the longitudinal positions of said end members;
  - end member support bodies retained in said openings in said tubular member, said support bodies having portions thereof extending into the interior of said

tubular member which overlie and are connected to a portion of said end members for transferring the load imposed on said end members to said tubular member;

a piston in said cylinder portion of said tubular member, said piston being longitudinally movable between said two end members;

a longitudinally extending working mandrel having its longitudinal axis aligned with the longitudinal axis of said tubular member, said mandrel being connected to said piston for longitudinal movement therewith, and having at least a portion of its longitudinal length disposed in said working portion of said tubular member;

a die means having a longitudinally extending billet chamber, said die means having a longitudinal passage therethrough to permit said working mandrel to enter into and pass through said billet chamber to perform the metal working of the billet;

means for supporting said die means in the working portion of said tubular member so that the axis of said billet chamber and said passage are aligned with the axis of said tubular member; and

supply means for supplying fluid to said cylinder portion for moving said piston longitudinally within said cylinder portion of said tubular member.

2. Apparatus for metal working of a billet comprising:

a longitudinally extending hollow tubular member;

a pair of axially spaced end members transversely supported in said tubular member to divide the interior of said tubular member into a cylinder portion located between said two end members and a working portion;

a piston in said cylinder portion of said tubular member, said piston being longitudinally movable between said two end members;

a longitudinally extending working mandrel having its longitudinal axis aligned with the longitudinal axis of said tubular member, said mandrel being connected to said piston for longitudinal movement therewith, and having at least a portion of its longitudinal length disposed in said working portion of said tubular member;

a die means, including a longitudinally extending billet chamber, a chamber end member, and a billet centering member, supported in the working portion of said tubular member, said die means having a longitudinal passage therethrough to permit said working mandrel to enter into and pass through said billet chamber to perform the metal working of the billet, said chamber end member being positioned in said longitudinal passage adjacent the end of said billet chamber which is remote from the end through which said working mandrel enters and said chamber end member being removable from said die means to allow for insertion of the billet into said billet chamber through said end remote from said entry end of said working mandrel, and said billet centering member being positioned at the end of said chamber end member adjacent to said billet chamber for centering the billet in said billet chamber as said working mandrel performs the metal working on the billet;

means for supporting said die means in the working portion of said tubular member so that the axis of said billet chamber and said longitudinal passage

are aligned with the axis of said tubular member, and said supporting means comprising means for retaining said chamber end member in position within said longitudinal passage of said die means, said means for retaining comprising two diametrically opposed openings in the side wall of said tubular member adjacent the longitudinal position of said chamber end member, a series of convolutions formed on laterally opposite sides of the longitudinal surface of said chamber end member, and two support bodies retained in said openings in said tubular member each of which has a series of convolutions on a portion thereof extending into the interior of said tubular member for mating engagement with said series of convolutions on said chamber end member; and

supply means for supplying fluid to said cylinder portion for moving said piston longitudinally within said cylinder portion of said tubular member.

3. The apparatus of claim 1 wherein said portions of said end member support bodies extend into said working portion of said tubular member.

4. The apparatus of claim 3 wherein said plurality of openings adjacent each of the longitudinal positions of said end members is two, each of said two openings being diametrically opposed to one another and communicating with said working portion of said tubular member; where each of said end member support bodies is a bar having a portion of its length intermediate its two ends positioned in said openings; and wherein there is provided connecting means at each of said longitudinal positions of said end members, each of said connecting means connecting together the ends of said bars positioned in said two diametrically opposed openings to retain said bars therein.

5. The apparatus of claim 2 wherein said working mandrel has a piercing end face for piercing the billet in said chamber and wherein said chamber end member has a longitudinal opening therethrough aligned with the longitudinal axis of said billet chamber, the size of the opening being such as to permit passage therethrough of said mandrel and to prevent passage therethrough of the billet.

6. The apparatus of claim 2 wherein said chamber end member comprises a first outer member which has a longitudinal bore therethrough, the size of said bore being smaller than the size of the opening in said billet chamber; and a second inner-member which fits into the bore of said first member, said second inner-member having a longitudinal bore therethrough which is smaller than said bore of said first outer member, said first and second members being individually and separably removable from said die means.

7. The apparatus of claim 2 wherein said working mandrel is connected to said piston by a longitudinally extending piston rod having one end attached to said piston and having the other end extending through one of said end members into said working portion of said tubular member, said mandrel being attached to said other end of said piston rod.

8. The apparatus of claim 7 further including a pair of longitudinal extending pull rods positioned in said working portion of said tubular member between said die means and the wall of said tubular member, each of said pull rods having one end attached to said piston rod and having the other end extending beyond said die means; and a cross member removably attached to both of said other ends of said pull rods, said cross member being engagable with a billet to force said billet into said billet chamber when said chamber end member is removed and said piston rod is moved in a direction away from said die means.

\* \* \* \* \*

**UNITED STATES PATENT OFFICE**  
**CERTIFICATE OF CORRECTION**

PATENT NO. : 3,851,516

DATED : December 3, 1974

INVENTOR(S) : Kenneth Bishop Garner and Lewis Shepherd Roberson

It is certified that error appears in the above-identified patent and that said Letters Patent are hereby corrected as shown below:

Column 2, lines 39 and 40	delete "leading" and insert --loading--.
Column 3, line 32	delete "member" and insert --members--.
Column 5, line 60	delete "10" and insert --110--.
Column 7, line 44	delete "and" and insert --an--.
Column 8, lines 13 and 14	delete "appropriate" and insert --appropriately--.
Column 10, line 26	delete "where" and insert --wherein--.
Column 10, line 28	following "in" insert --one of--.
Column 10, line 57	delete "longitudinal" and insert --longitudinally--.
Column 3, line 66	delete "dry" and insert --by--.
Column 6, line 63	delete "pirecing" and insert --piercing--.
Column 7, line 60	delete "manual" and insert --mandrel--.
Column 7, line 63	delete "arrangment" and insert --arrangement--.

**Signed and Sealed this**

*twentieth Day of April 1976*

[SEAL]

*Attest:*

**RUTH C. MASON**  
*Attesting Officer*

**C. MARSHALL DANN**  
*Commissioner of Patents and Trademarks*