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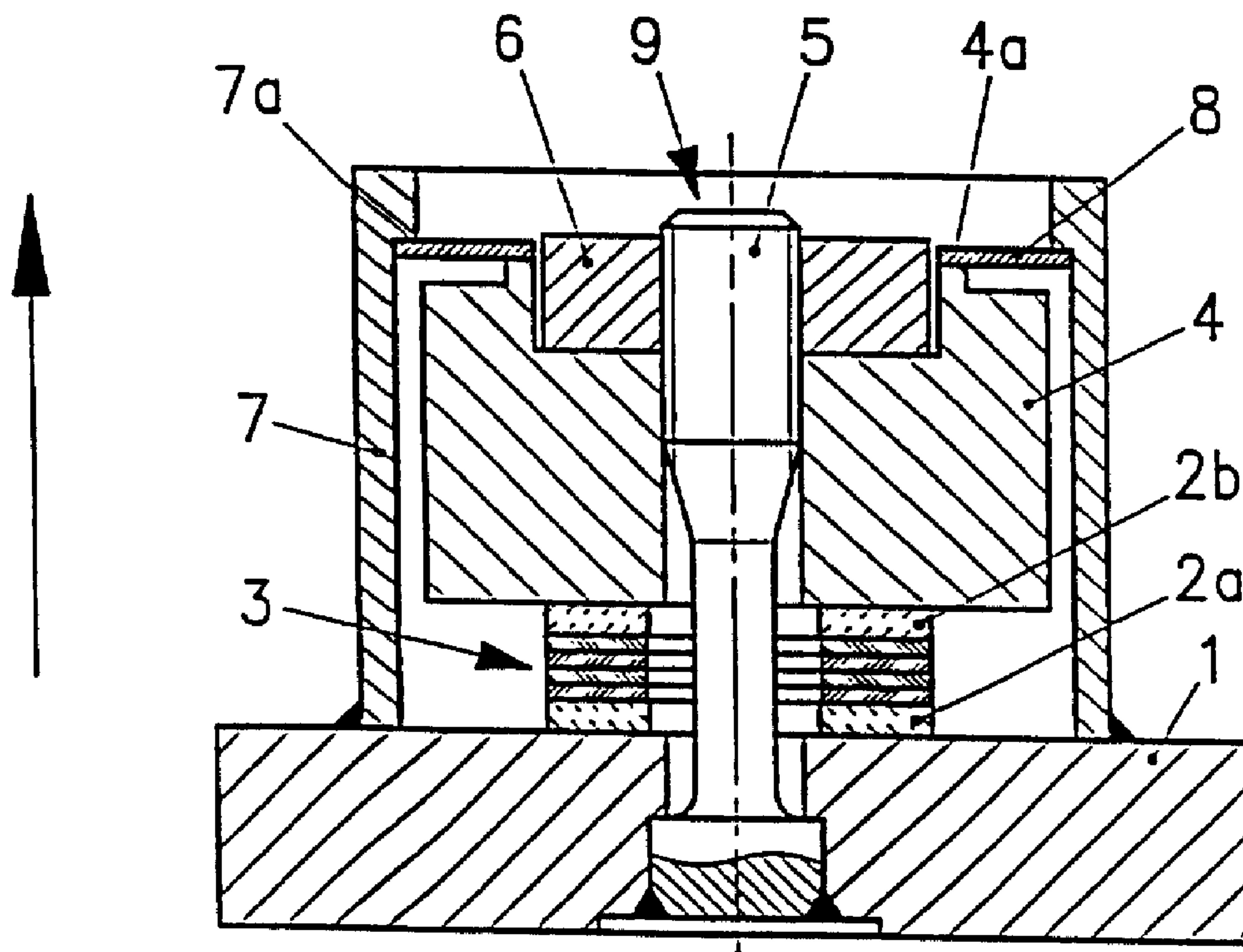
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(54) Titre : ACCELEROMETRE PIEZOELECTRIQUE AVEC ELEMENT DE STABILISATION LATERALE

(54) Title: PIEZOELECTRIC ACCELEROMETER WITH Laterally Stabilizing Element



(57) Abrégé/Abstract:

The seismic mass (4) of an accelerometer that comprises at least one electromechanical transducer is stabilized in the direction perpendicular to the sensible axis by at least one laterally stabilizing element (8) arranged between the electromechanical transducer (9) and a structural element (7). This stabilization eliminates or at least drastically reduces any distortions of the measurement results provoked by transversal accelerations.



Abstract of the Disclosure

The seismic mass (4) of an accelerometer that comprises at least one electromechanical transducer is stabilized in the direction perpendicular to the sensible axis by at least one
5 laterally stabilizing element (8) arranged between the electromechanical transducer (9) and a structural element (7). This stabilization eliminates or at least drastically reduces any distortions of the measurement results provoked by transversal accelerations.

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(Figure 2a)

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PIEZOELECTRIC ACCELEROMETER WITH Laterally Stabilizing ELEMENT

Field of the Invention

The present invention relates to an accelerometer comprising
5 a supporting base and at least one electro-mechanical
transducer constituted of at least one sensitive element,
each of said at least one sensitive element comprising at
least one piezoelectric element, the whole being arranged
between said supporting base and a seismic mass.

10 Background of the Invention

It is known to use piezoelectric accelerometers for
measuring the vibrations of rotating machines such as gas
turbines or jet engines. Among the known basic principles
used for the design of accelerometers, there are two that
15 are the most frequently used, namely, the shear mode design
and the compression mode design. The compression mode
designs can be split in two subgroups. A first subgroup
using a pure compression of the piezoelectric material (i.e.
 d_{33} , d_{11} or d_{22} mode) whereas a second subgroup uses the
20 transverse compression of the piezoelectric material (i.e.
 d_{31} , d_{21} , d_{32} , d_{23} , d_{13} , d_{12} mode). These two basic designs use
at least one seismic mass which, under the effect of the
acceleration generated by the vibrations, acts upon one or
more piezoelectric elements.

25 In the shear mode accelerometer design, a deformation of the
piezoelectric elements produced by a shear force is used for
producing an electric charge, whereas in the compression
mode accelerometer design, a longitudinal compression of the

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piezoelectric elements in the direction of the sensitive axis produces an electric charge.

Each one of these two basic accelerometer designs has advantages and limitations for the design engineer, but also
5 in the practical implementation. Shear mode accelerometers can be very small devices for a given sensitivity, but have a lower resonance frequency. Compression mode accelerometers are known to be suitable for applications which require a high resonance frequency. In view of the piezoelectric
10 properties of certain materials, design engineers prefer compression mode accelerometers for high temperature applications.

Serious problems may arise when piezoelectric accelerometers are excited by high vibration levels in the transverse axis,
15 i.e. in a direction normal to the sensitive axis. This problem is encountered more specifically when the transversal vibration excitation frequency coincides with the transversal resonance of the transducer whereby the movement is amplified. Under these conditions, the bending
20 stress on the piezoelectric elements can produce a significant noise in the output signal to such an extent that this output signal is substantially corrupted. In the past, this phenomenon has very often not been correctly understood, and the results could be interpreted as a
25 pyroelectric effect or an effect of a sensitivity to strain of the supporting base (base strain).

Summary of the Invention

The main aim of the invention is to overcome the above-indicated problems and to provide an accelerometer having a

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structure that eliminates or at least substantially reduces the influence of transversal accelerations on the accuracy of the measurement results.

Another important and essential aim of the invention is in particular to provide a compression mode accelerometer
5 having a structure that eliminates or at least substantially reduces the influence of transversal accelerations on the accuracy of vibration signal provided by the accelerometer.

According to the invention the above aims are attained with
10 an accelerometer comprising:

- a supporting base,
- at least one electromechanical transducer each of which is fastened on said supporting base, and each of which includes at least one seismic mass, and at least one
15 sensitive element, each such sensitive element comprising at least one piezoelectric element,
- a structural element which is rigidly connected to said supporting base, and
- at least one laterally stabilizing element which is in
20 mechanical contact with said at least one electromechanical transducer and with said structural element.

In a preferred embodiment of an accelerometer according to the invention, the laterally stabilizing element is substantially more rigid in radial than in axial direction.
25 The above mentioned structural element may constitute a part of an accelerometer housing which housing is rigidly fastened to the supporting base.

Further special or preferred features of the present invention will be disclosed and emphasized in the following

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detailed description of embodiments thereof, and in the dependent claims.

The invention is applicable to all basic designs of accelerometers but it is useful in particular for
5 compression mode accelerometers (the basic structure of this type of accelerometer is shown by FIG. 1).

Brief Description of the Drawings

FIG. 1 schematically shows the basic structure of a
typical, known piezoelectric accelerometer in
10 compression mode,

FIG. 2a, 2b show two embodiments of compression mode
accelerometers according to the invention,

FIG. 2c, 2d, 2e show variants of the embodiments shown by
FIGS. 2a and 2b.

15 FIG. 3 shows the basic structure of a typical known
piezoelectric accelerometer in shear mode, and

FIG. 4a, 4b show two embodiments of shear mode
accelerometers according to the invention,

FIG. 4c, 4d, 4e show variants of the embodiments shown by
20 FIGS. 4a and 4b.

FIG. 5 shows a perspective view of an embodiment of
the type described with reference to FIG. 2d.

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Detailed Description of the Invention

Fig. 1 shows the basic structure of a known, compression mode accelerometer. This accelerometer comprises an electromechanical transducer 9 which is fastened on a supporting base 1. The electromechanical transducer 9 comprises a sensitive element 3, constituted by one or more piezoelectric elements mounted between two insulating elements 2a, 2b, a seismic mass 4 mounted on upper insulating element 2b, a central bolt 5 and a nut 6. The electrodes (not shown for the sake of clarity) which serve for collecting electric charge are disposed between the piezoelectric elements. The electromechanical transducer 9 comprising the components just mentioned is fastened on the supporting base 1 by means of the central bolt 5 and the nut 6. The bolt 5 is prestressed for avoiding any displacement of the components of the electromechanical transducer 9 under all conditions of use. Instead of the prestressed bolt 5, the components of the electromechanical transducer may also be glued, welded or fastened in any other suitable way. The electromechanical transducer 9 is mounted in a housing 7' which is rigidly fastened to the supporting base 1, e.g. by welding.

An accelerometer of the type represented in Fig. 1 is fastened through its supporting base 1 on the surface of an object by means of a fixation device (not shown) in order to measure vibrations of that surface.

In all enclosed drawings the direction of the sensitive axis of the accelerometer is indicated by an arrow represented on the left of each of Figures 1 to 4e, and said arrow is parallel to said sensitive axis.

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Under the action of vibrations in the direction of the sensitive axis of the accelerometer, the seismic mass 4, situated at the free end of the electromechanical transducer 9, generates compression and decompression forces against the supporting base 1 and also on the piezoelectric elements which, in turn, generate electrical charges representative of the measurement signal. The accelerometer functions correctly as long as the decompression forces, including a engineering margin, remain weaker than the prestressing force of the central bolt.

Under the action of vibrations in the direction perpendicular to the sensitive axis, the free end of the electromechanical transducer 9 will bend laterally in the direction of the excitation. The maximum bending stress experienced by the piezoelectric elements may be very high and is typically higher than the compression stress under the same magnitude of acceleration in the direction of the sensitive axis. This bending effect may lead to an important distortion of the vibration signal, in particular when the frequency of the vibration excitation coincides with the bending resonance frequency of the assembly.

Two embodiments of compression mode accelerometers according to the present invention, are represented in FIG. 2a and 2b. These embodiments typically comprise the same parts as those shown in the known embodiment according to FIG. 1, but comprise an additional part, namely, a laterally stabilizing element 8 which is in contact with the electromechanical transducer 9 and a structural element 7. This structural element is rigidly fastened to the supporting base 1. In the most simple embodiment, the structural element 7 is part of

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the housing 7' of the accelerometer, and housing 7' is also rigidly fastened to the supporting base 1.

In the preferred embodiment of FIG. 2a, the laterally stabilizing element 8 is a simple cut-out from a metal sheet. An outer end part of this cut-out rests on a shoulder 7a machined in structural element 7 and an inner part of the latter cut-out touches the electromechanical transducer 9, for example at end part 4a of the seismic mass 4. The axial positions of resting points 7a and 4a on which the cut-out rests are selected in such a manner that the metal sheet which constitutes the laterally stabilizing element 8 is slightly deformed. This ensures that the laterally stabilizing element 8 is always in slight contact with shoulder 7a and resting point 4a.

15 In the preferred embodiment shown by FIG. 2b, the laterally stabilizing element 8 is welded or connected by any other means to the electromechanical transducer 9 and to structural element 7, respectively, for example to end part 4b of the seismic mass 4 and to point 7b of the structural element 7.

The laterally stabilizing element 8 may have the shape of a disk (represented in FIG. 2c), a ring (represented in FIG. 2d) or at least one bar (represented in FIG. 2e) that is arranged in radial direction and which connects
25 electromechanical transducer 9 to structural element 7.

The preferred embodiments just described ensure a lateral stabilization of the electromechanical transducer as well as an additional damping of the bending resonance and allow to obtain a vibration signal that is free from noise under all
30 conditions. It is important that laterally stabilizing

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element 8 is as rigid as possible in radial direction, but is as flexible as possible in axial direction, namely, in the sensitive direction of the piezoelectric elements. In this manner, lateral stabilization of the electromechanical transducer is obtained without influencing the axial behavior of the assembly.

Furthermore, all influences arriving through housing 7', such as acoustic noise, are not converted into an alteration of the vibration signal, because the laterally stabilizing element 8 cannot transmit any significant forces in axial direction.

FIG. 3 schematically shows the basic structure of a typical, known shear mode piezoelectric accelerometer. This accelerometer comprises an electromechanical transducer 20 which is fastened on a supporting base 11. The electromechanical transducer 20 comprises one or more sensitive elements 13a and 13b, each of them being formed by at least one piezoelectric elements mounted between two insulating elements 12a, 14a and 12b, 14b, respectively, seismic masses 15a and 15b mounted outside second insulating elements 14a and 14b, a central bolt 17 and a nut 18. The electrodes (not shown for sake of clarity) which serve for collecting electric charge are disposed between the piezoelectric elements. The electromechanical transducer 20 comprising the components just mentioned is fastened on the supporting base 11 by means of the central bolt 17 and the nut 18. The bolt 17 is prestressed for avoiding any displacement of the components of the electromechanical transducer 20 under all conditions of use. Instead of the prestressed bolt 17, the components of the electromechanical transducer 20 may also be glued, welded or fastened in any

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other suitable way. The electromechanical transducer 20 is mounted in a housing 19' which is rigidly fastened to the supporting base 11, e.g. by welding.

An accelerometer of the type represented in Fig. 3 is
5 fastened through its supporting base 11 on the surface of an object by means of a fixation device (not shown) in order to measure vibrations of that surface.

Under the action of vibrations in the direction of the sensitive axis of the accelerometer, the seismic masses 15a
10 and 15b generate shear forces on the piezoelectric elements which, in turn, generate electric charges representative of the measurement signal.

Under the action of vibrations in the direction perpendicular to the sensitive axis, the free end of the
15 electromechanical transducer 20 will laterally in the direction of the excitation. The maximum bending stress experienced by the piezoelectric elements, may be very high and may lead to an important distortion of the vibration signal, in particular when the frequency of the vibration
20 excitation coincides with the bending resonance frequency of the assembly.

Two embodiments of shear mode accelerometers according to the present invention are represented in FIG. 4a and 4b. These embodiments typically comprise the same parts as those
25 shown in the known embodiment according to FIG. 3, but comprise an additional part, namely, a laterally stabilizing element 21 which is in contact with the electromechanical transducer 20 and a structural element 19. This structural element is rigidly fastened to supporting base 11. In the
30 most simple embodiment, the structural element 19 is part of

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the housing 19' of the accelerometer, and housing 19' is also rigidly fastened to supporting base 11.

In the preferred embodiment shown in FIG. 4a, the laterally stabilizing element 21 is a simple cut-out from a metal sheet. An outer end part of this cut-out rests on a shoulder 19a machined in structural element 19 and a central part of that cut-out touches the electromechanical transducer 20, for example at its end part 20a. The axial positions of resting points 19a and 20a on which the cut-out rests are selected in such a manner that the metal sheet which constitutes the laterally stabilizing element 21 is slightly deformed. This ensures that the laterally stabilizing element 21 is always in slight contact with shoulder 19a and resting point 20a.

15 In the preferred embodiment of FIG. 4b, the laterally stabilizing element 21 is welded or connected by any other means to the electromechanical transducer 20 and to the structural element 19, respectively, for example to its end part 20b and to point 19b of the structural element 19.

20 The laterally stabilizing 21 may be formed by a disk (represented in FIG. 4c), a ring (represented in FIG. 4d) or at least one bar (represented in FIG. 4e) that is arranged substantially in radial direction and connects the electromechanical transducer 20 to the structural element 19.

25

Structural element 7, 19 has preferably the shape of a symmetric cylinder (see FIG. 2a, 2b, 4a, 4b, 5) having a diameter, and thus a moment of inertia, greater than that of

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the bolt 5, 17 in order to obtain a symmetrical and efficient stabilizing effect.

The preferred embodiments just described ensure a lateral stabilization of the electromechanical transducer as well as
5 an additional damping of the bending resonance and allow to obtain a vibration signal that is free from noise under all conditions. It is important that the laterally stabilizing element 21 is as rigid as possible in radial direction, but is as flexible as possible in axial direction, namely, in
10 the sensitive direction of the piezoelectric elements. In this manner, the lateral stabilization of the electromechanical transducer is obtained without influencing the axial behavior of the assembly.

Furthermore, all influences arriving through the housing
15 19', such as acoustic noise, are not converted into an alteration of the vibration signal, because the laterally stabilizing element 21 cannot transmit any significant forces in axial direction.

The present invention may thus be also applied with benefit
20 to piezoelectric accelerometers that are constructed for working in the shear mode and allows to avoid or at least to radically reduce the alteration effect of the vibration signal mentioned above.

Modifications and alternative embodiments of the above
25 described embodiments of an accelerometer according to the invention will be apparent to those skilled in the art in view of the foregoing description. Accordingly, this description is to be construed as illustrative only and is for the purpose of teaching those skilled in the art the

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best mode of carrying out the invention. Details of the
accelerometer may be varied substantially without departing
from the spirit of the invention and the exclusive use of
all modifications which come within the scope of the
5 appended claims is reserved.

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What is claimed is:

1. An accelerometer comprising
 - (a) a supporting base,
 - (b) at least one electromechanical transducer each of
5 which is fastened on said supporting base, and each of which includes
 - (b.1) at least one seismic mass, and
 - (b.2) at least one sensitive element, each such
sensitive element comprising at least one piezoelectric
10 element,
 - (c) a structural element which is rigidly connected to
said supporting base, and
 - (d) at least one laterally stabilizing element which is
in mechanical contact with said at least one
15 electromechanical transducer and with said structural
element.
2. The accelerometer according to claim 1, wherein said at
least one laterally stabilizing element is substantially
more rigid in radial direction than in axial direction.
- 20 3. The accelerometer according to claim 1, wherein said
structural element forms part of the housing of the
accelerometer, said housing being rigidly fastened to said
supporting base.
- 25 4. The accelerometer according to claim 1, wherein an end
of said at least one laterally stabilizing element rests on
a shoulder of said structural element, and the other end of
said at least one laterally stabilizing element touches said
electromechanical transducer.

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5. The accelerometer according to claim 1, wherein the axial positions of the resting points of said at least one laterally stabilizing element are selected such that said at least one laterally stabilizing element is slightly deformed
5 in order to ensure that it is maintained in slight mechanical contact with said electromechanical transducer.

6. The accelerometer according to claim 1, wherein said at least one laterally stabilizing element is a simple cut-out from a metal sheet.

10 7. The accelerometer according to claim 1, wherein said at least one laterally stabilizing element has the shape of a ring.

8. The accelerometer according to claim 1, wherein said at least one laterally stabilizing element has the shape of a
15 disk.

9. The accelerometer according to claim 1, wherein said at least one laterally stabilizing element is formed by one or more bars arranged in substantially radial direction.

10. The accelerometer according to claim 1, wherein an end
20 of said at least one laterally stabilizing element is welded or fastened by any other means to said structural element, and the other end of said at least one laterally stabilizing element is welded or fastened by any other means to said electromechanical transducer.

25 11. The accelerometer according to claim 1, wherein said at least one laterally stabilizing element has a cylindrical shape and surrounds said at least one seismic mass.

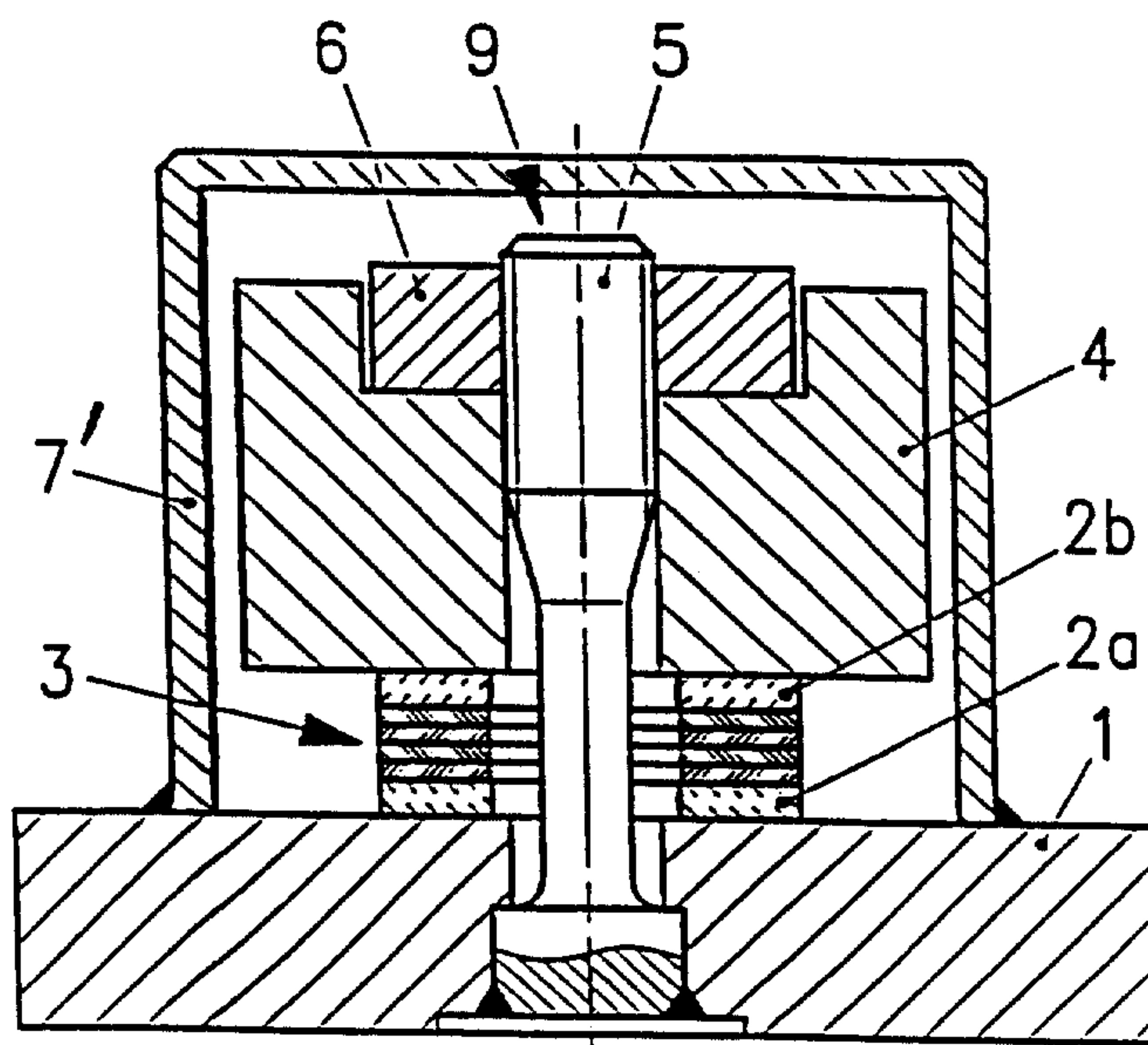


Fig. 1

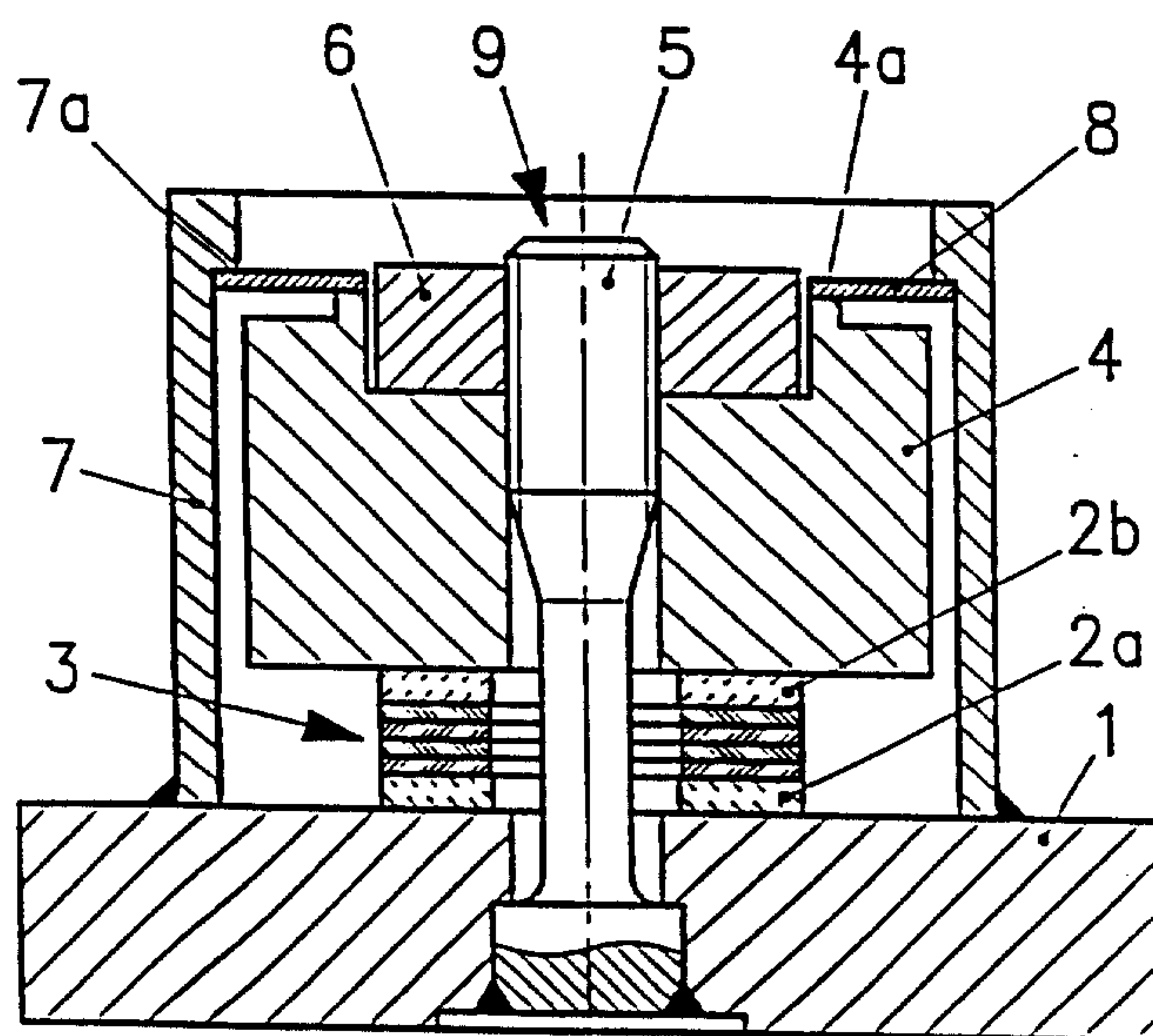


Fig. 2a

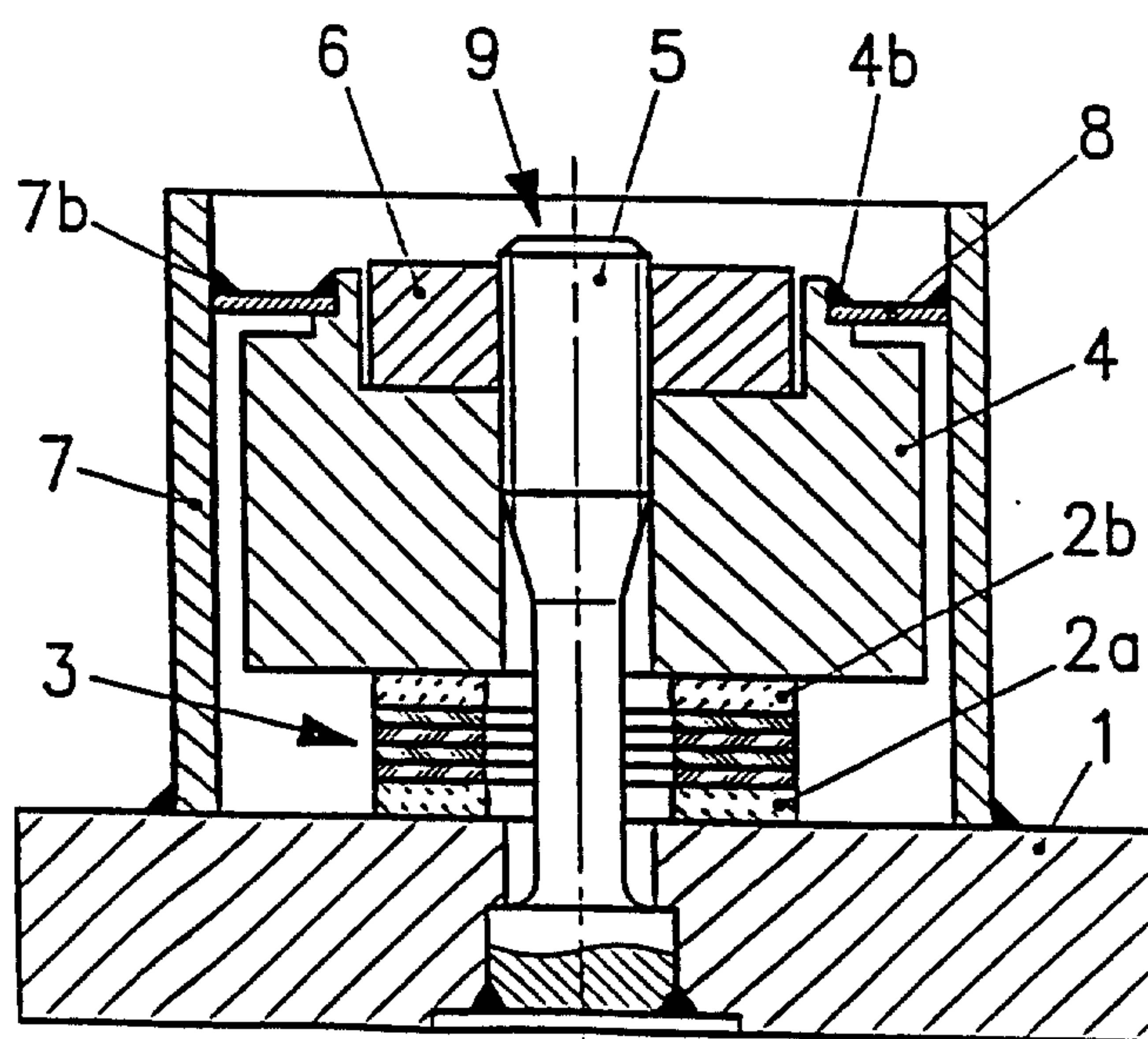


Fig. 2b

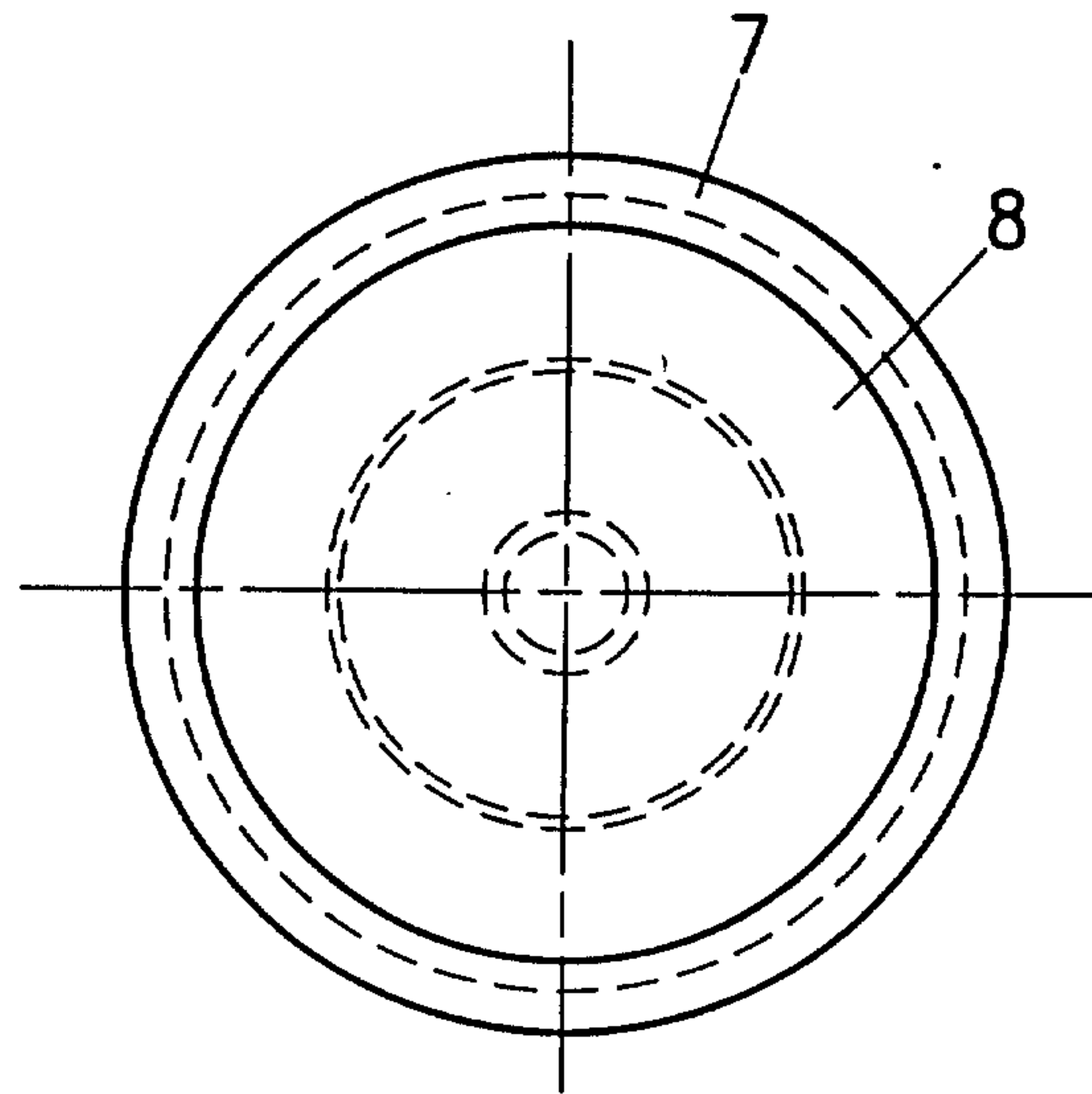


Fig. 2c

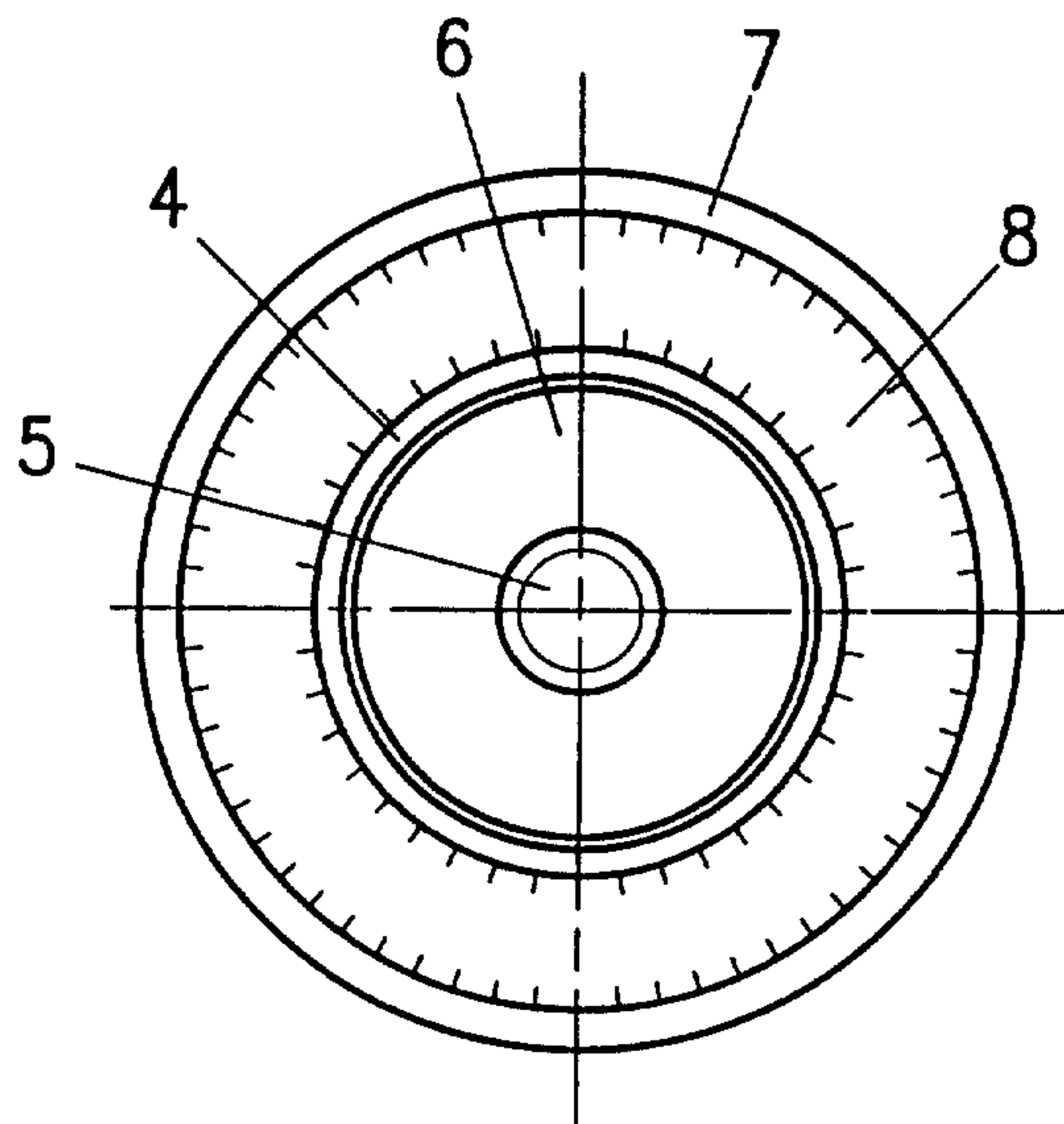


Fig. 2d

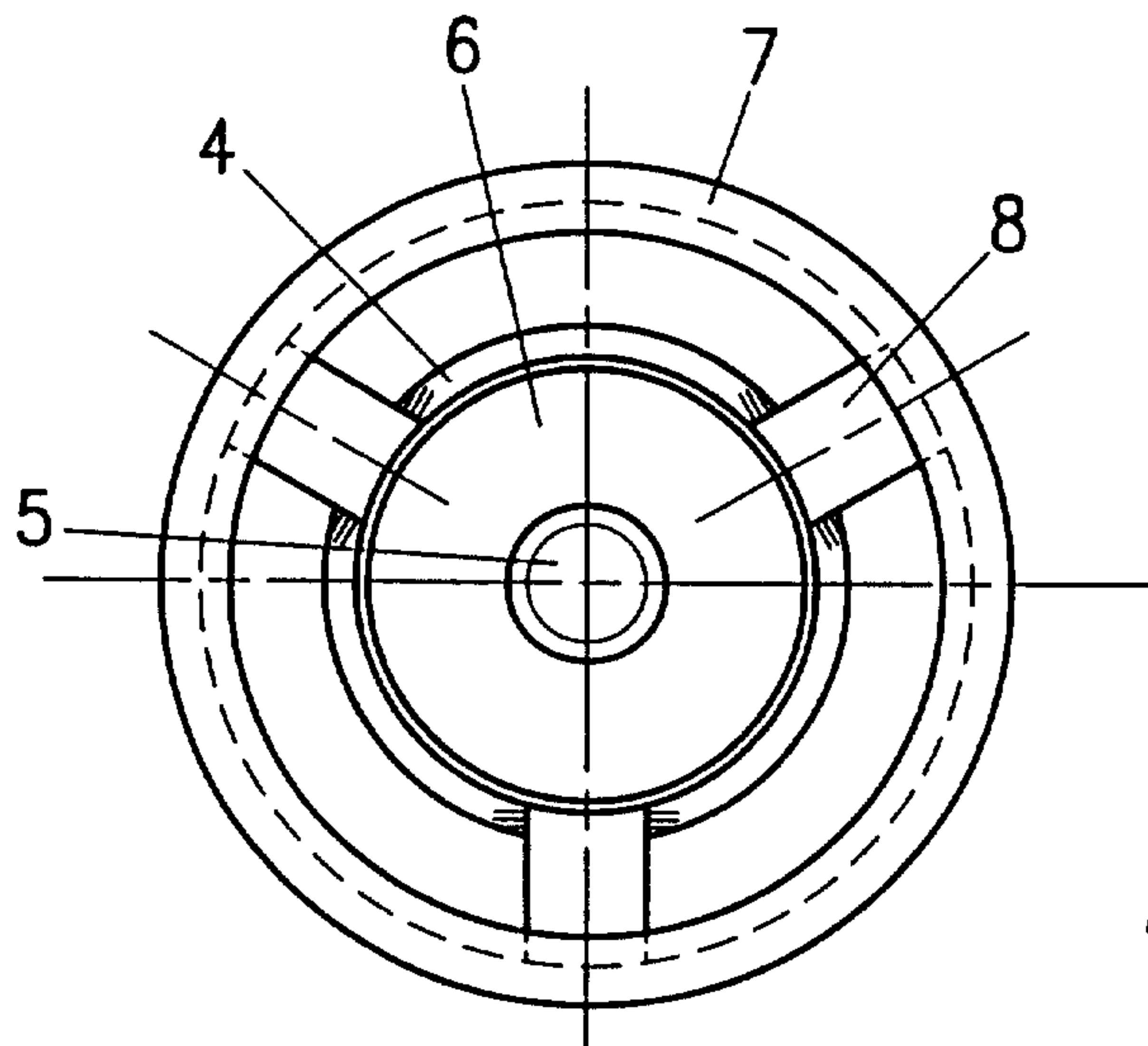


Fig. 2e

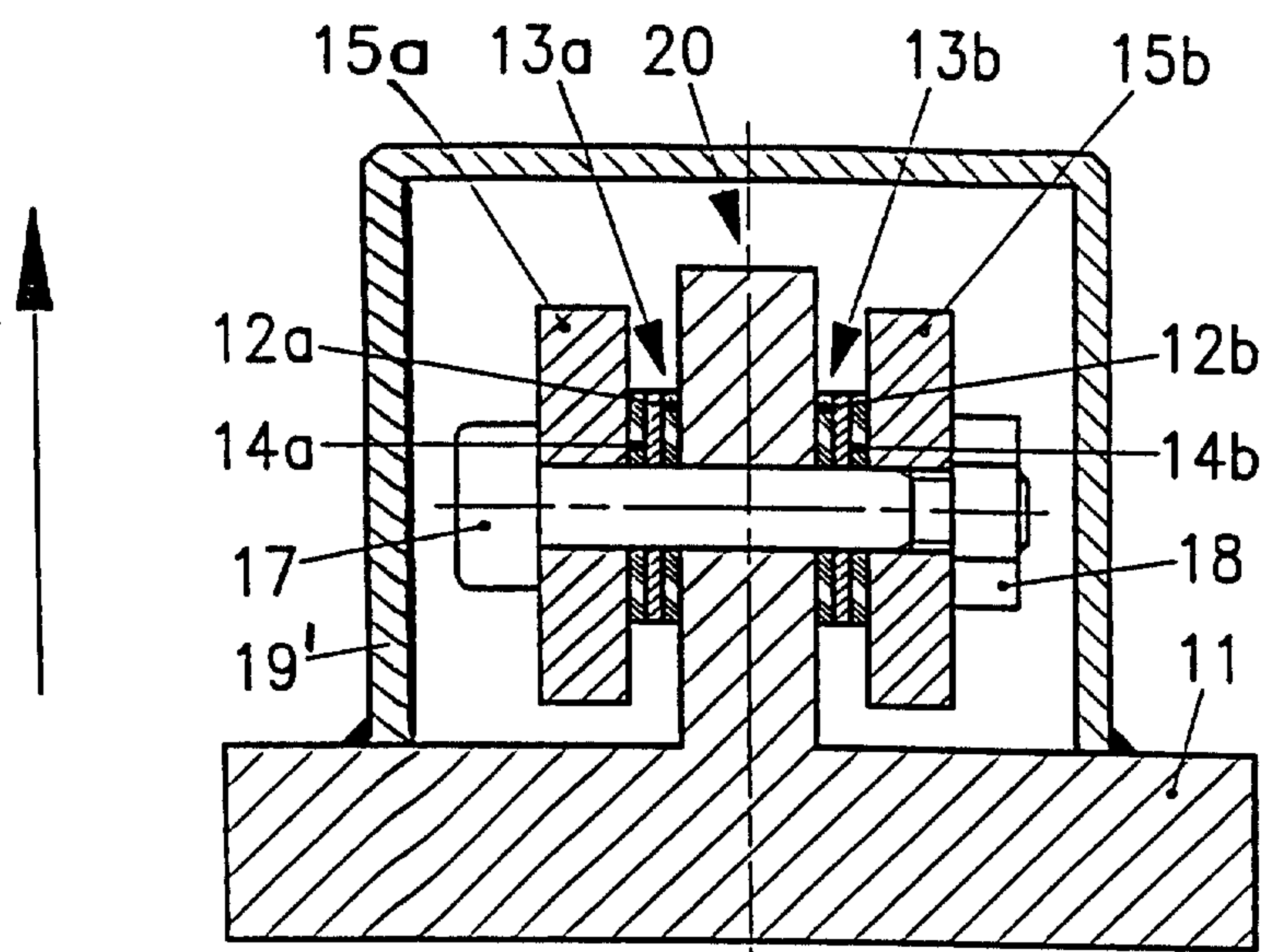


Fig. 3

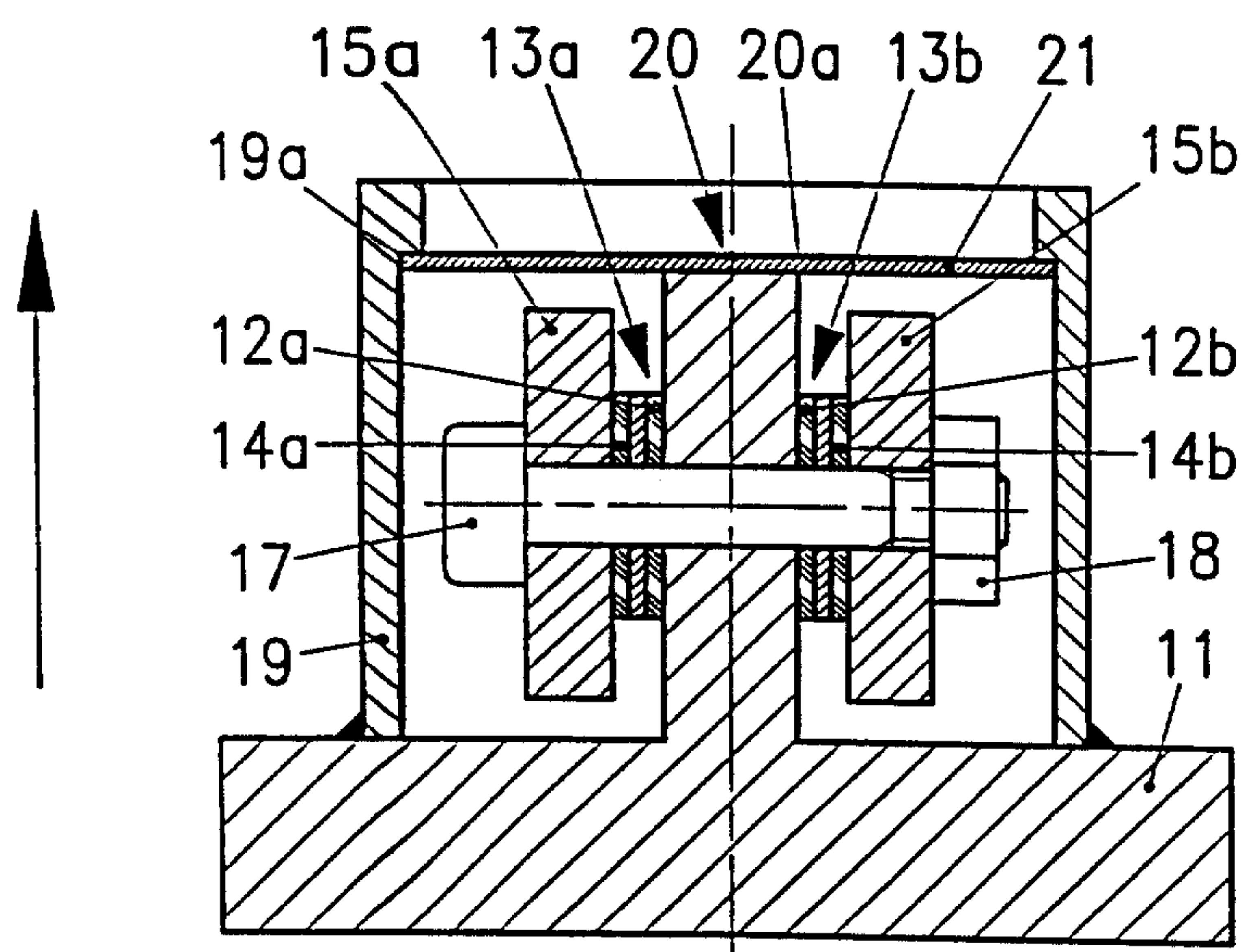


Fig. 4a

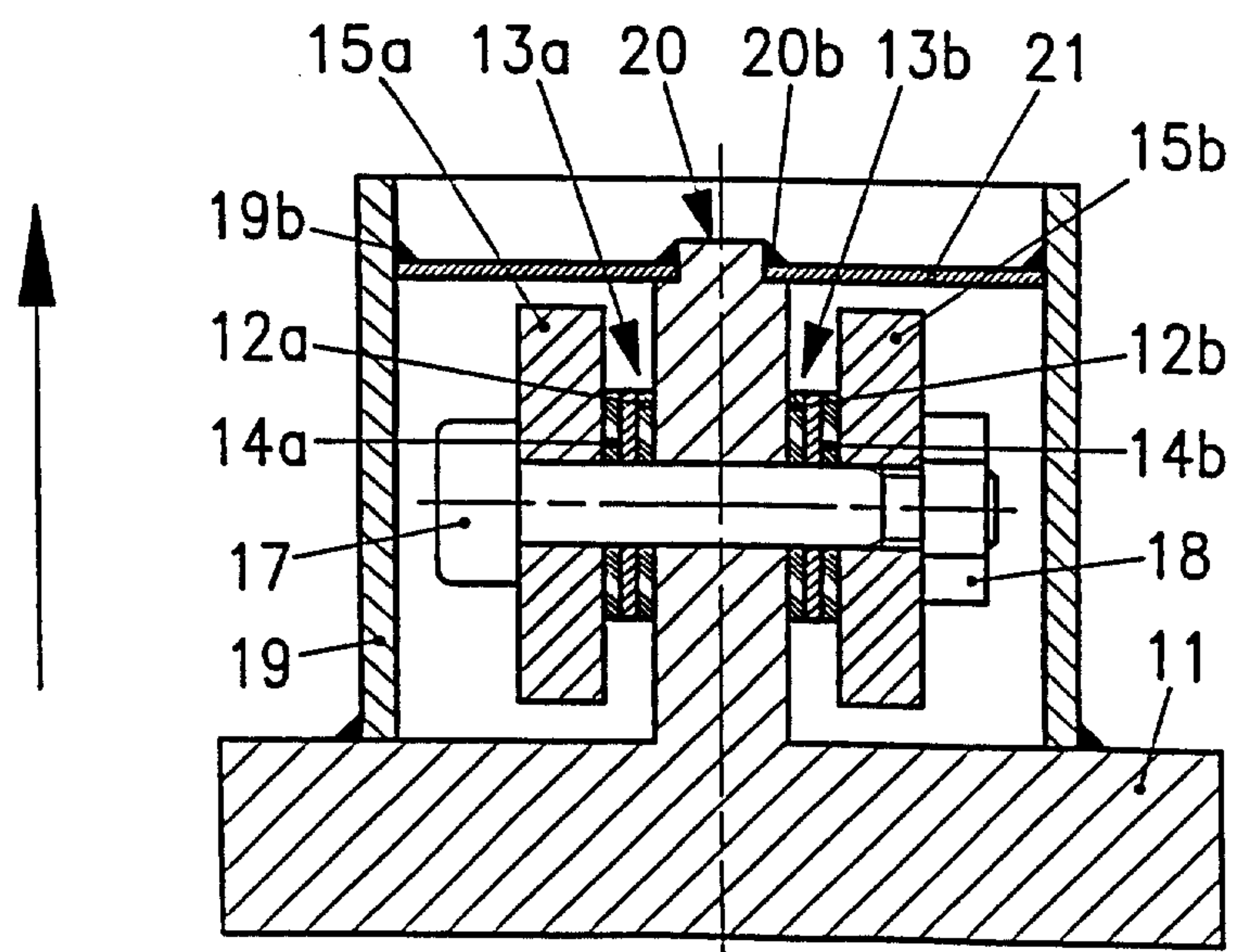


Fig. 4b

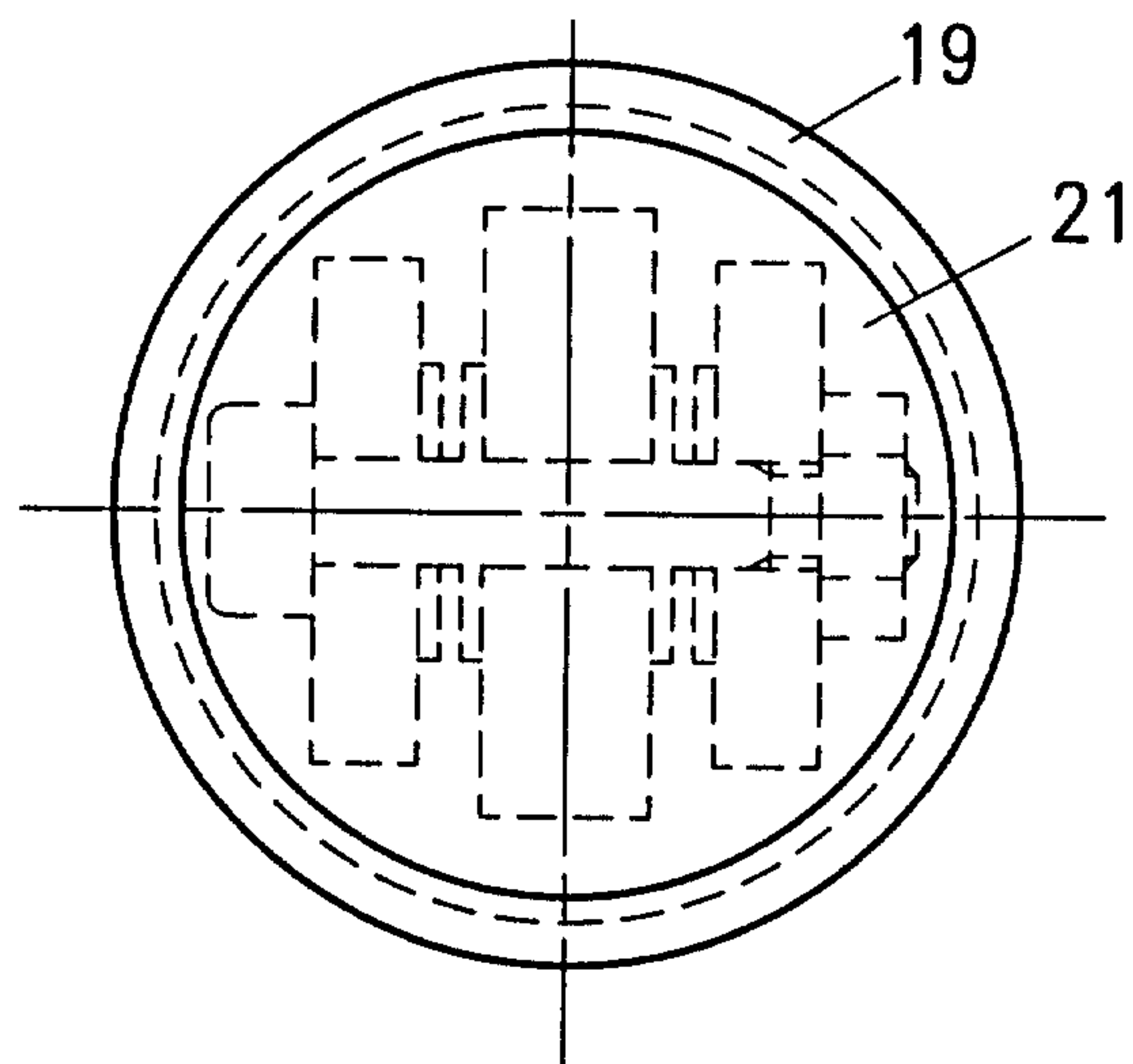


Fig. 4c

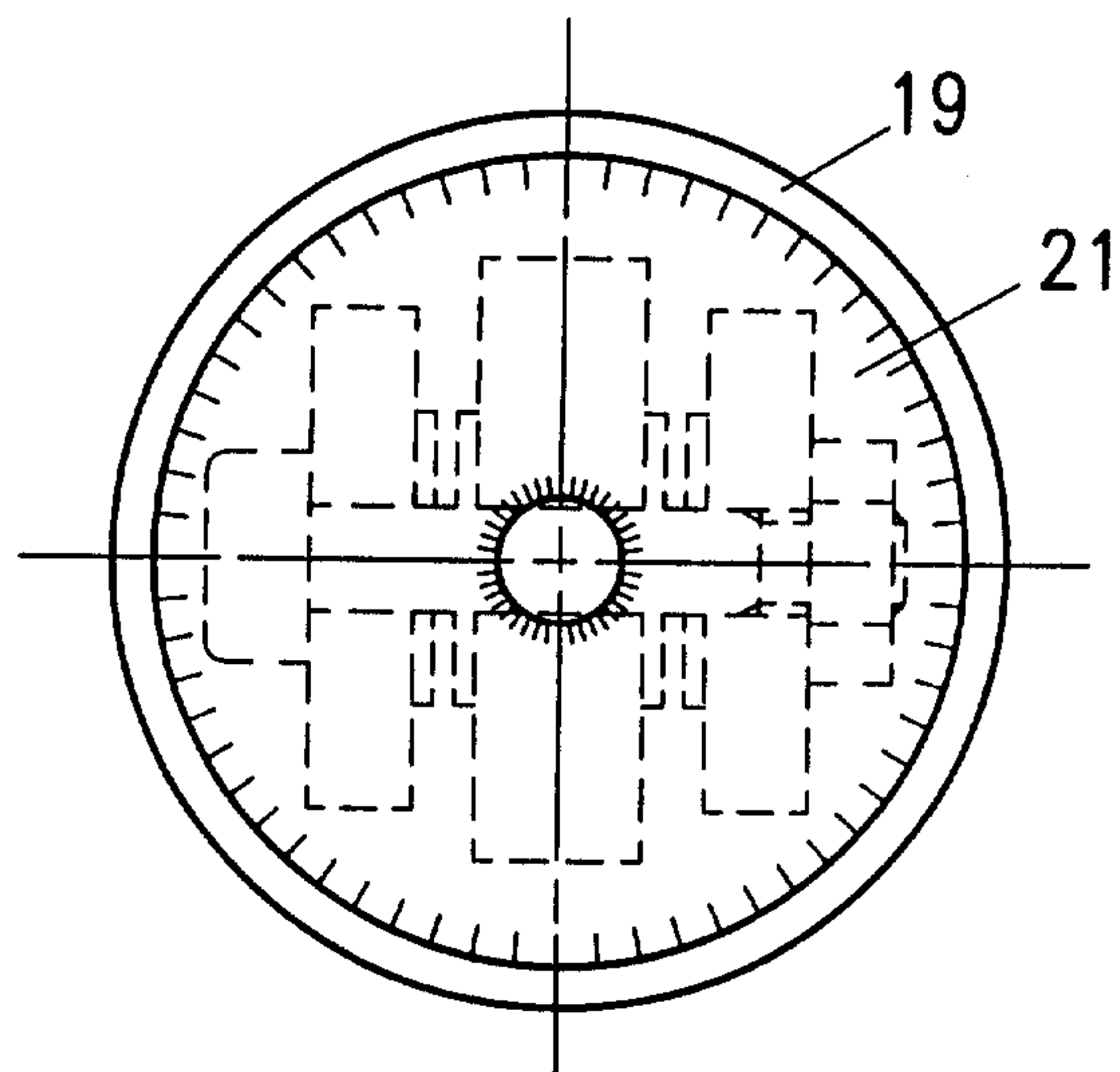


Fig. 4d

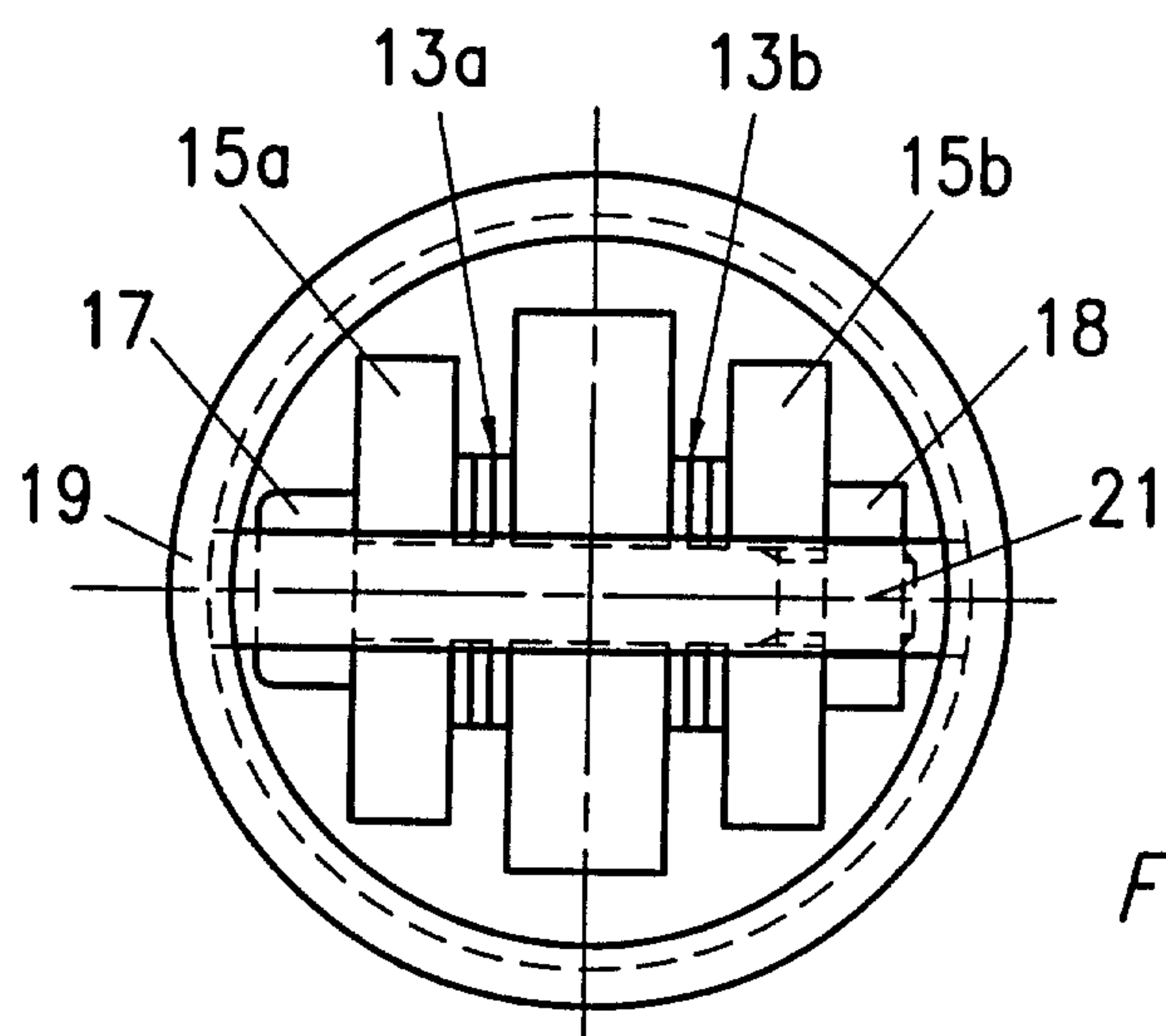


Fig. 4e

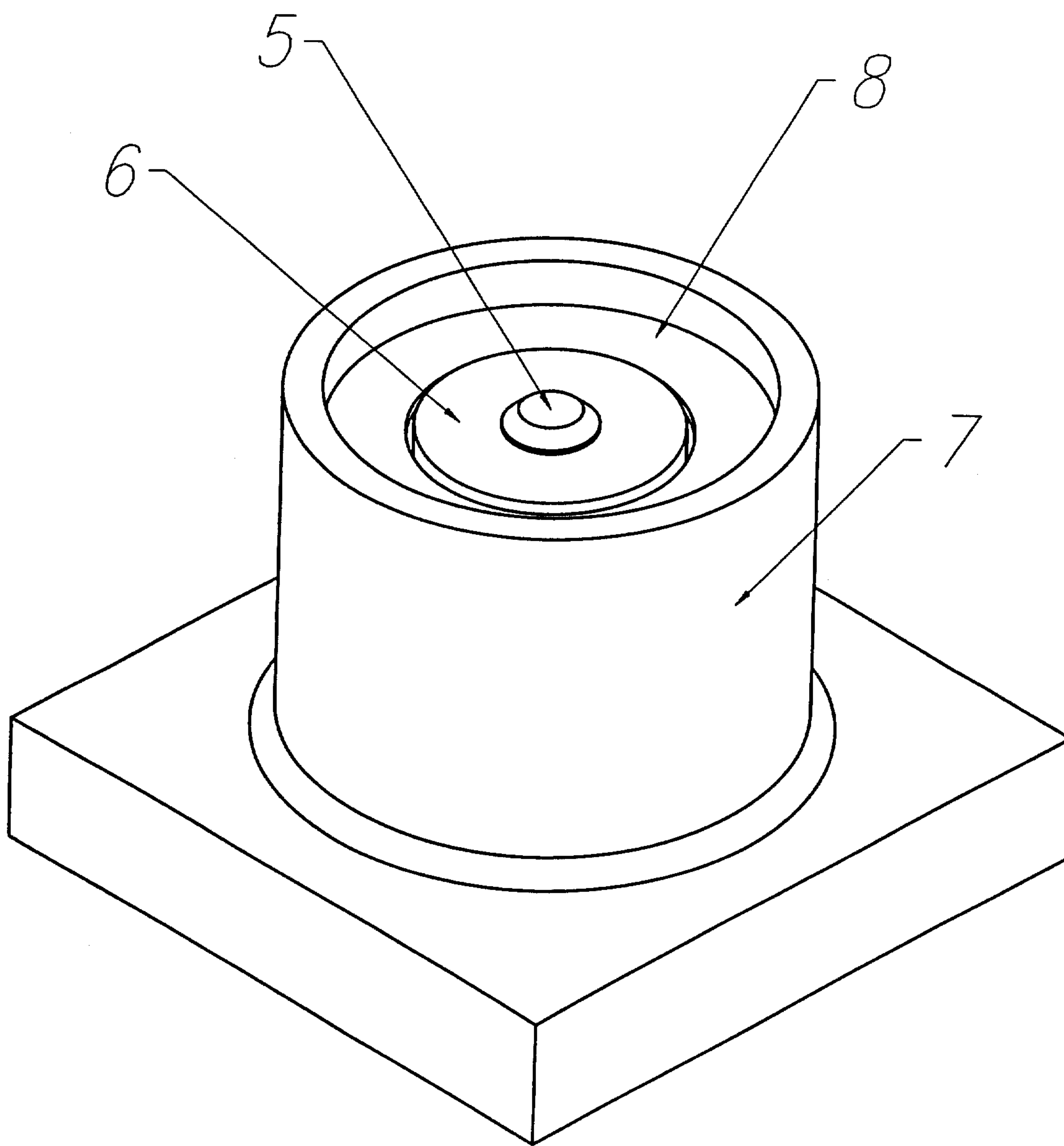


Fig. 5

