

[54] **ROCKER ARM**[75] Inventor: **Fred G. Loon**, Taylor, Mich.[73] Assignee: **Ford Motor Company**, Dearborn, Mich.[22] Filed: **May 15, 1973**[21] Appl. No.: **360,489**[52] U.S. Cl. **123/90.27**, 123/90.34, 123/90.36,
123/90.44, 184/6.9[51] Int. Cl. **F01L 1/02**, F01L 1/04[58] Field of Search 123/90.36, 90.34, 90.27,
123/90.44; 184/6.9

[56]

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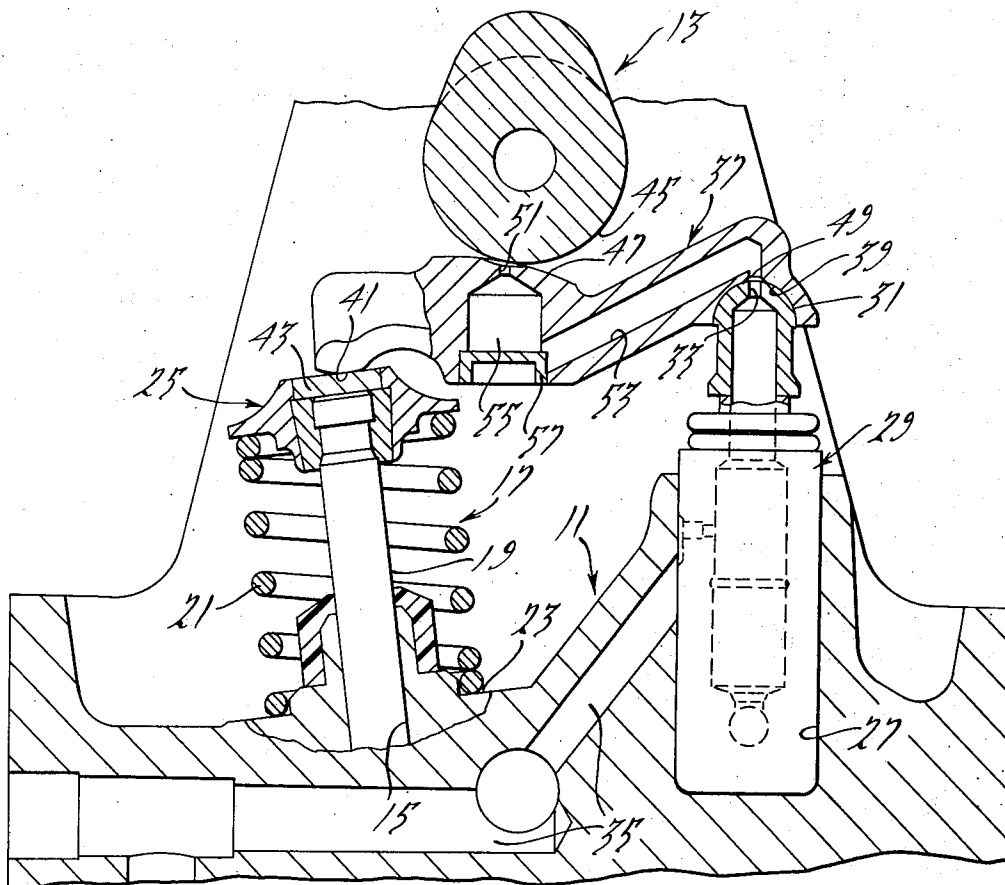
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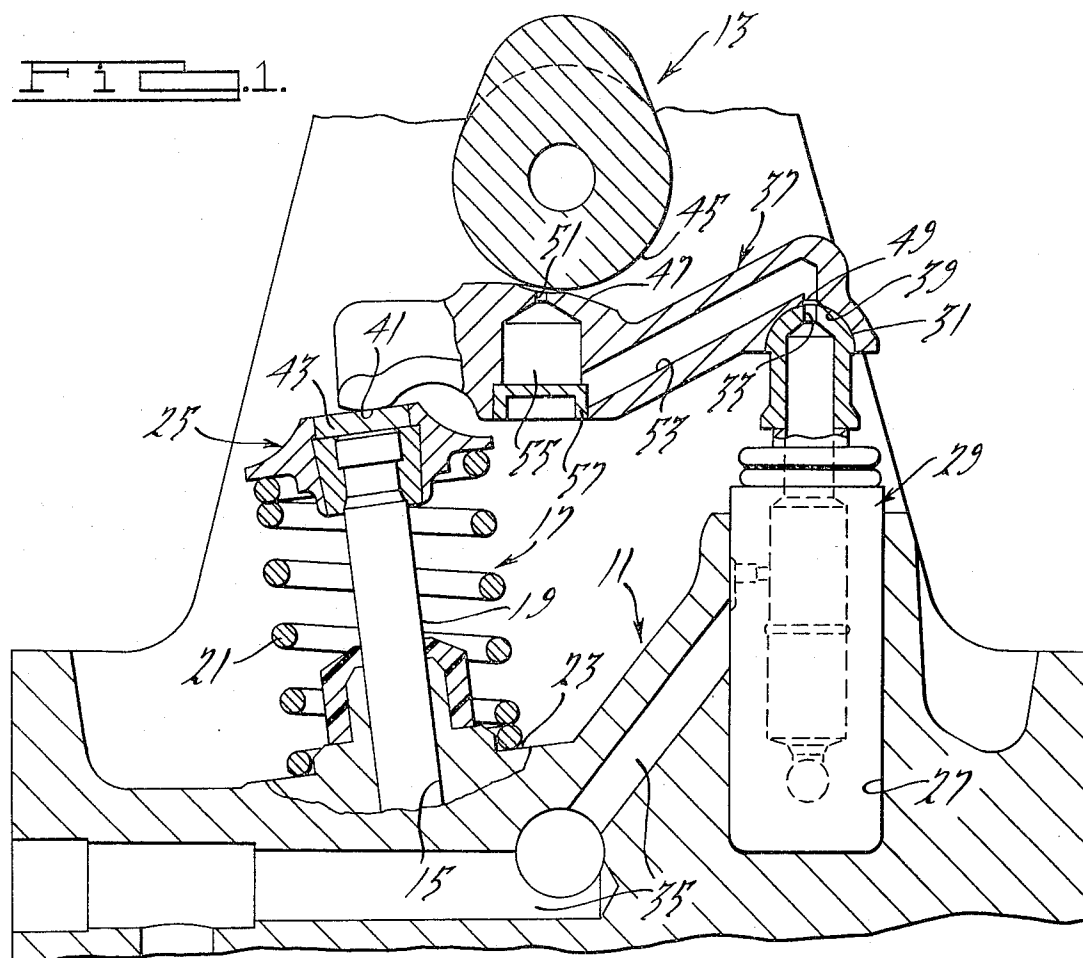
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ABSTRACT

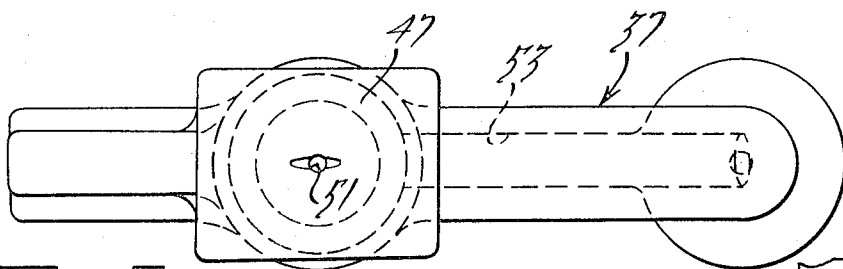
A rocker arm for an internal combustion engine having overhead camshaft. The rocker arm includes an internal lubrication passage connecting the fulcrum socket and the cam pad. During engine operation oil is provided under pressure to the rocker arm from the fulcrum member and is discharged at the cam pad directly beneath the cam surface. The passage may include an oil reservoir beneath the cam pad which retains a predetermined quantity of oil when the engine is not in operation and which provides essentially instantaneous lubrication to the cam pad and cam surface upon start-up of the engine.

7 Claims, 4 Drawing Figures

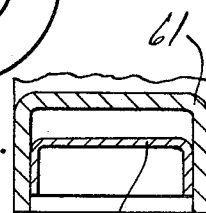
A horizontal beam is shown with a triangular load of intensity 1 acting downwards over a section of length 1. To the right of this section, there is a rectangular load of intensity 1 acting downwards over a section of length 1. The beam is supported by a pin support at the left end and a roller support at the right end.



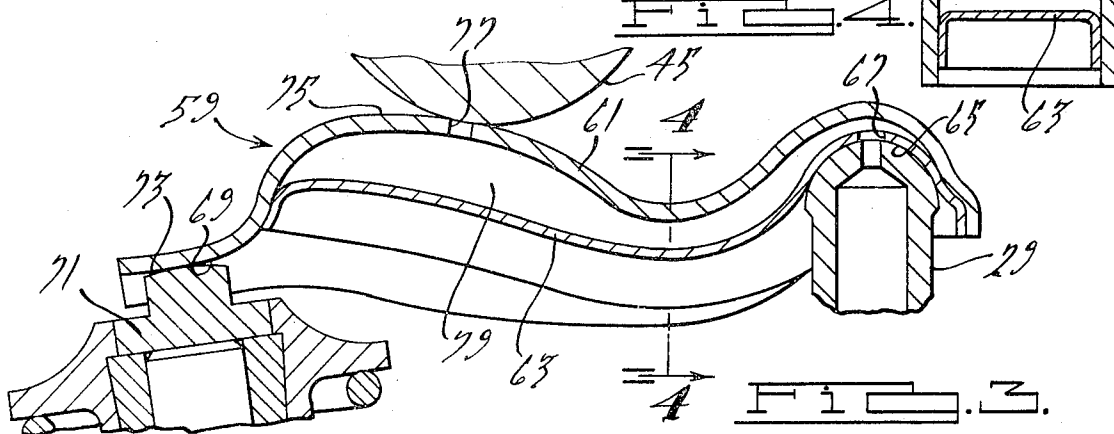
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A horizontal beam is shown with a triangular load increasing linearly from left to right. A vertical support is located at the right end of the beam.



A horizontal beam is shown with a triangular load of intensity 100 lb/ft acting downwards over a 6 ft segment starting from the left end. A point load of 100 lb acts downwards at the right end of the beam. The beam is supported by a pin support at the left end and a roller support at the right end. The total length of the beam is 12 ft.



1 ROCKER ARM

BACKGROUND AND SUMMARY OF THE INVENTION

An internal combustion engine having an overhead camshaft generally incorporates a system to provide lubrication to the mating cam and rocker arm surfaces. In one such prior art system, oil is provided through the camshaft mounting supports to an axial passage formed within the camshaft and is discharged from ports opening on the cam surfaces. Another prior art lubrication system employs a rocker arm having an internal lubrication passage receiving oil from the rocker arm fulcrum member and discharging the oil from a jet against a cam surface prior to its engagement with the rocker arm pad.

This invention provides a cam lubrication system which receives lubricating oil from the rocker arm fulcrum member and which overcomes certain disadvantages associated with both the prior art systems identified previously. This invention provides a rocker arm and cam surface lubrication system which instantaneously supplies oil to the sliding surfaces upon initial engine start-up and which significantly reduces oil delivery delay often experienced in prior art systems. This invention also provides a lubrication system in which oil is provided to the cam and rocker arm interface during 360° of camshaft rotation. Furthermore, this invention provides a rocker arm incorporating a lubrication system which functions identically for either direction of rotation of the camshaft. This invention also provides a rocker arm having an oil supply port that delivers oil at a location on the cam surface to most effectively carry away and dissipate heat generated by the mating surfaces. Finally, this invention provides a rocker arm construction which may be either cast or fabricated from stamped elements and which is economical to produce and reliable in operation.

A rocker arm constructed in accordance with this invention has a first end pivotally connected to a fulcrum member mounted to the cylinder head. The other end of the rocker arm is engageable with the protruding end of the valve stem or valve assembly. A camshaft is slidably engageable with a cam pad formed on the rocker arm at a point between the two previously mentioned ends. An oil passage is formed in the rocker arm having an inlet receiving oil from the fulcrum member and an outlet on the cam pad immediately beneath the engaged portion of the cam surface. The passage within the rocker arm may include an enlarged reservoir positioned immediately beneath the outlet to the cam surface.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a cross sectional view with portions in elevation of a valve actuating mechanism including a rocker arm constructed in accordance with this invention.

FIG. 2 is an enlarged top view of the rocker arm.

FIG. 3 is a cross sectional view of an alternate embodiment of the invention.

FIG. 4 is a cross sectional view taken along the line 4—4 of FIG. 3.

DETAILED DESCRIPTIONS OF PREFERRED EMBODIMENTS

Reference numeral 11 of the drawings refers to a portion of a cylinder head of an internal combustion en-

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gine having an overhead camshaft 13. Reciprocally received within a bore 15 formed in the cylinder head is an intake or exhaust valve assembly 17 having a stem 19 protruding upwardly from the cylinder head. The valve assembly 17 includes a retainer assembly 25 which comprises a coiled compression spring 21 between itself and a seat 23 formed in the cylinder head surface. Spaced from bore 15 and coplanar therewith is a second bore 27 which receives the rocker arm fulcrum member or pivot support member 29. The fulcrum member may be a threaded stud or a hydraulic lash adjuster as shown in FIG. 1. The protruding end of the fulcrum member is a convex spherical surface 31 having an oil outlet port 33 formed therein. The oil outlet port is connected through the fulcrum member to a source of pressurized oil 35 within the cylinder head.

Bridging the end of the valve stem 19 and the end 31 of the fulcrum member 29 is a rocker arm 37. The rocker arm has a concave spherical surface or socket 39 formed in the underside of one end corresponding in size to the spherical surface 31 of the fulcrum member. Formed on the underside of the other end of the rocker arm is a slightly convex surface or pad 41 that engages the upper element 43 of the valve assembly.

Mounted to the cylinder head 11 to rotate about an axis perpendicular to the plane formed by the axis of the valve stem 19 and the fulcrum member 29 is the camshaft 13. The camshaft has a cam surface 45 which slidably engages the pad 47 formed on the upper side of the rocker arm.

Communicating with the oil outlet port 33 of the fulcrum member 29 is an oil inlet 49 formed in the socket 39 of the rocker arm. An oil discharge port 51 is formed centrally within the cam pad 47 of the rocker arm immediately beneath the engaged portion of the cam surface 45. Interconnecting the discharge port 51 with the inlet 49 is a passage 53. Immediately beneath the discharge port and the cam pad and forming a portion of passage 53 is an enlarged reservoir 55. A plug 57 closes the bottom of the reservoir. The inlet 49 is of sufficient height so that when the engine is not in operation the reservoir retains a substantial quantity of lubricating oil.

It may be seen from the drawings that the rocker arm 37 is a casting having the oil ports and passages formed by boring. An alternate embodiment is shown in FIGS. 3 and 4 of the drawings and comprises a functionally equivalent stamped version of the rocker arm. The stamped rocker arm 59 includes an outer member 61 and an inner member 63 welded in place within the enclosure of the outer member. The spherical socket 65 engaging the fulcrum member 29 is stamped into the inner member and includes a centrally located inlet port 67. The valve stem engaging portion 69 comprises a channel which fits over a raised portion 73 of valve assembly element 71. The cam pad 75 is a slightly convex portion of the upper surface of the outer member 61 as shown in FIG. 3. An oil discharge port 77 is formed centrally in the cam pad along the line of engagement with the cam surface 45. The volume between the inner member 63 and the outer member 61 defines a passage connecting the inlet 67 and the discharge port 77, and further defines an oil reservoir 79 positioned immediately beneath the discharge port. Similarly to the rocker arm of FIG. 1, the inlet 67 is positioned so that the reservoir 79 retains a substantial

portion of lubricating oil when the engine is not in operation.

In either embodiment, fulcrum member may be provided with a check valve (not shown) to prevent siphoning of oil from the interior of the rocker arm to the fulcrum member 29 and cylinder head passages 35.

Upon initial starting of the engine, the oil in the reservoir 55 or 79 is instantaneously splashed or vibrated through discharge port 51 or 77, respectively, onto the cam surface 45. Thus, it may be seen that the cam and rocker arm engaging portions receive lubricating oil immediately without delay while the oil is pumped through the lubricating system through the pivot stud and into the cam lubricating passage. It may also be seen that lubricating oil is provided to the cam and rocker arm at times when it is most likely to be devoid of lubrication and when it is most susceptible to damage by scratching.

Modifications and alterations will occur to those skilled in the art which are included within the scope of the following claims.

I claim:

1. In an internal combustion engine having a cylinder head, a poppet valve assembly having a stem reciprocally mounted in said cylinder head, a camshaft rotatably mounted to said cylinder head, and a fulcrum member mounted to said cylinder head,

a rocker arm having a first end portion connected to said fulcrum member and a second end portion engageable with the end of said valve assembly, said camshaft slidably engaging said rocker arm at an area intermediate said end portions, a passage formed in said rocker arm, said passage having inlet means at said first end portion of said rocker arm, said passage having outlet means formed in and located at the camshaft engaging area of said rocker arm, said rocker arm receiving oil from said fulcrum member for discharge from said outlet directly against said camshaft.

2. In an internal combustion engine according to claim 1, said passage including an enlarged reservoir within said rocker arm.

3. In an internal combustion engine according to claim 2, said reservoir being positioned immediately upstream of said outlet means.

4. In an internal combustion engine having a cylinder head, a camshaft rotatably mounted above said cylinder head, a poppet valve assembly reciprocally mounted within the cylinder head, said valve assembly having an end portion protruding from said cylinder head, a fulcrum member having a portion received within said cylinder head and a portion protruding therefrom having a convex spherical end,

a rocker arm extending from the end said fulcrum member to said valve assembly, said rocker arm having a spherical socket at one end positionable over and corresponding to the convex spherical end of said fulcrum member,

an outlet port formed in said convex spherical end of said fulcrum member,

an inlet port formed in said spherical socket of said rocker arm, said inlet port being in communication with said outlet port,

said rocker arm having a first pad at its other end engageable with the end portion of said valve assembly,

a second pad formed on a surface of said rocker arm facing oppositely of said first pad, said second pad being intermediate said first pad and said spherical socket,

said camshaft including a cam surface slidably contacting a portion of said second pad,

a discharge port formed in said portion of said second pad contacting said cam surface,

passage means formed within said rocker arm interconnecting said inlet and discharge ports of said rocker arm.

5. In an internal combustion engine according to claim 4,

said passage including a reservoir immediately beneath the discharge port of the rocker arm.

6. In an internal combustion engine according to claim 5,

said passage including said reservoir having a configuration which precludes said reservoir from complete draining through said inlet port of the rocker arm when the engine is not in operation.

7. In an internal combustion engine according to claim 5,

a portion of said reservoir being located below a horizontal line through the inlet port of the rocker arm to prevent drainage from said reservoir portion when the engine is not in operation.

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