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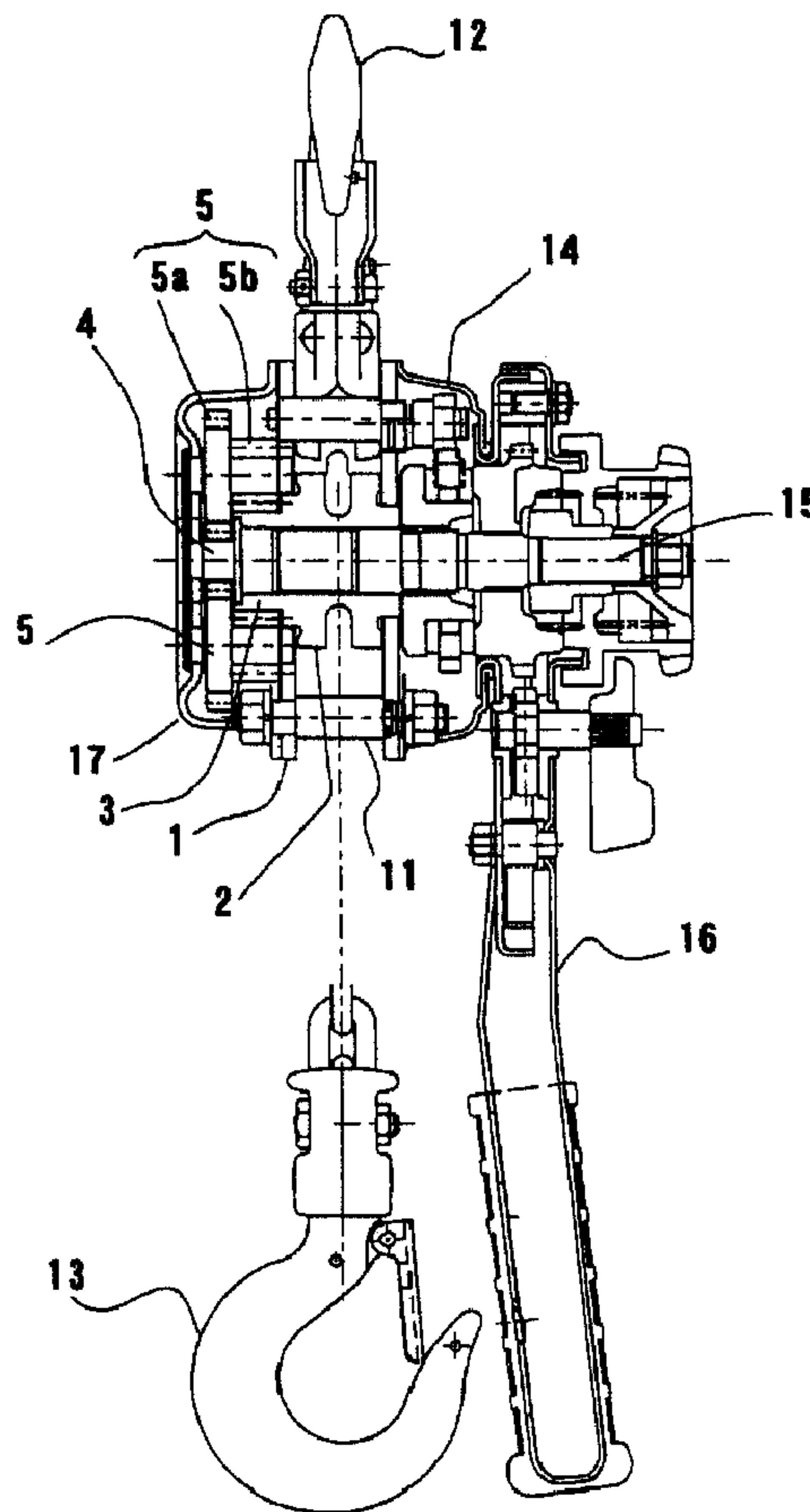
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(54) Title: HOISTING/PULLING DEVICE



(57) Abrégé/Abstract:

In a hoisting/pulling device comprising a pinion gear 4, reduction gears 5a which are meshed with the pinion gear 4 and are rotatably mounted in a frame 1, and a load sheave which is interlocked with reduction gears 5b and is rotatably mounted in a frame,

(57) **Abrégé(suite)/Abstract(continued):**

reduction gear bearings 8 are contiguously formed with a load sheave bearing 7 which is formed in the center of the frame 1. By making the distance between the axes of the reduction gears small and by forming the load gear and the load sheave integrally, it becomes possible to provide the hoisting/pulling device which can make the whole device compact and light-weighted and can reduce the number of parts and man-hours for machining.

ABSTRACT

In a hoisting/pulling device comprising a pinion gear 4, reduction gears 5a which are meshed with the pinion gear 4 and are rotatably mounted in a frame 1, and a load sheave which is interlocked with reduction gears 5b and is rotatably mounted in a frame, reduction gear bearings 8 are contiguously formed with a load sheave bearing 7 which is formed in the center of the frame 1. By making the distance between the axes of the reduction gears small and by forming the load gear and the load sheave integrally, it becomes possible to provide the hoisting/pulling device which can make the whole device compact and light-weighted and can reduce the number of parts and man-hours for machining.

Description

HOISTING/PULLING DEVICE

5 Technical Field

The present invention relates to a hoisting/pulling device, and more particularly to a hoisting/pulling device which can make the whole device compact and light-weighted and can reduce the number of parts and the man-
10 hours for machining and assembling.

Background Art

The support structure of reduction gears and the structure of a load gear in a conventional
15 hoisting/pulling device are explained in conjunction with Fig. 5 and Fig. 6.

In these drawings, numeral 21 indicates a frame for a load sheave bearing and reduction gear bearings, numeral 22 indicates a load sheave, numeral 23 indicates
20 a load gear, numeral 24 indicates a pinion gear, numeral 25 indicates reduction gears a which are meshed with the pinion gear 24, numeral 26 indicates reduction gears b which are mounted coaxially with the reduction gears a, numeral 27 indicates the load sheave bearing which is
25 formed in the center of the frame 21, numeral 28 indicates bearings for reduction gear shafts which are disposed at positions outside the load sheave bearing 27 and a given distance away from the load sheave bearing 27, numeral 29 indicates an upper hook mounting hole,
30 numeral 30 indicates holes for mounting stud bolts, numeral 31 indicates stud bolts, numeral 32 indicates an upper hook, numeral 34 indicates a mechanical brake which is comprised of brake plates, a brake support, a ratchet

wheel and a pawl, numeral 35 indicates a pinion shaft, numeral 36 indicates a manipulation lever, and numeral 37 indicates a casing.

In the above-mentioned conventional device, as shown in Fig. 6, the bearing 28 for the reduction gear b which are formed in the frame 21 are, to ensure the support strength of the frame, disposed at a position a given distance A away from the load sheave bearing 27. Accordingly, the distance between the axes of the pinion gear and the reduction gear and the distance between the axes of the reduction gear and the load gear respectively become large and the diameters of respective gears become large. Eventually, the casing which covers these gears becomes large. Further, since the distance between the axes of the reduction gears 25, 26 is large, it is necessary to insert the load gear 23 between the reduction gear 2 and the load sheave 22 to transmit the rotation of the reduction gear 2 to the load sheave. Further, since the load gear has the diameter which is larger than that of a bearing portion of the load sheave, it is necessary to assemble the pinion shaft having the load gear and the pinion gear after assembling the load sheave into the bearing portion of the frame.

25 Disclosure of the Invention

As described above, in the conventional hoisting/pulling device, the distance between the axes of the reduction gear and the pinion gear and the distance between the axes of the reduction gear and the load gear are large and respective gears are large-sized. This prevents the hoisting/pulling device from being miniaturized. Further, it is necessary to assemble the pinion shaft having the load gear and the pinion gear

after assembling the load sheave into the bearing of the frame and hence, it is difficult to preliminarily integrally form the load gear and the load sheave. The present invention has been made to solve these drawbacks
 5 of the conventional device and provides a hoisting/pulling device which can make the whole device compact and light-weighted and can reduce the number of parts and the man-hours for machining and assembling by making the distance between the axes of the reduction
 10 gears small and by making the outer diameter of the load gear equal to or smaller than the diameter of the bearing portion of the load sheave, and by integrally forming the load gear on the load sheave preliminarily.

15 Brief Description of the Drawings

Fig. 1 is a longitudinal cross-sectional view of a hoisting/pulling device of the present invention.

Fig. 2 is a plan view of a frame shown in Fig. 1.

Fig. 3 is a longitudinal cross-sectional view of a
 20 hoisting/pulling device of another embodiment of the present invention.

Fig. 4 is a plan view of a frame shown in Fig. 3.

Fig. 5 is a longitudinal cross-sectional view of a conventional hoisting/pulling device.

25 Fig. 6 is a plan view of a conventional frame.

Numerals in the drawings indicate followings.

- | | | |
|----|----|----------------------------------|
| | 1 | frame |
| | 1a | frame for load sheave bearing |
| | 1b | frame for reduction gear bearing |
| 30 | 2 | load sheave |
| | 3 | load gear |
| | 4 | pinion gear |
| | 5a | reduction gear a |

- 5b reduction gear b
- 7 load sheave bearing
- 7a load sheave bearing
- 8 reduction gear bearing
- 5 8a reduction gear bearing
- 9 upper hook mounting hole
- 10 stud bolt mounting hole
- 11 stud bolt
- 12 upper hook
- 10 13 lower hook
- 14 mechanical brake
- 15 pinion shaft
- 16 manipulation lever
- 17 casing

15

Best Mode for Carrying Out the Invention

The present invention has been made in view of the drawbacks that the above-mentioned conventional technique has and is directed to a hoisting/pulling device which includes a pinion gear, reduction gears which are interlocked with the pinion gear, and a load sheave which is driven by way of a load gear which is interlocked with the reduction gears, wherein the load gear has a diameter equal to or smaller than an outer diameter of a bearing portion of the load sheave and the reduction gears are directly meshed with the load gear.

According to the present invention, by making the diameter of the load gear equal to or smaller than the outer diameter of the load sheave, the distance between the axes of the load gear and the reduction gears which are meshed with the load gear can be made small so that the reduction gears can be rotatably mounted in the bearings which are contiguously formed with the load

sheave bearing whereby an advantageous effect that the device can be made compact and light-weighted and the number of parts, the man-hours for machining and assembling can be reduced is obtained.

5 The invention described in claim 1 of the present invention is directed to a hoisting/pulling device which includes a pinion gear, reduction gears which are interlocked with the pinion gear, and a load sheave which is driven by way of a load gear which is interlocked with
10 the reduction gear, wherein the load gear has a diameter equal to or smaller than an outer diameter of a bearing portion of a load sheave and the reduction gears are directly meshed with the load gear. By making the load gear have the diameter equal to or smaller than the outer
15 diameter of the bearing portion of the load sheave, it becomes possible to integrally form the load sheave and the load gear so that the load sheave can be assembled into the frame in the state that the load sheave is integrally assembled with the load gear whereby the
20 number of parts and the man-hours for machining and assembling can be reduced thus giving rise to an advantageous effect that a low-cost hoisting/pulling device can be provided.

 The invention described in claim 2 is characterized
25 by forming the load sheave and the load gear by an integral molding. This provision functions to reduce the number of parts and the man-hours for machining and assembling.

 The invention described in claim 3 is characterized
30 by rotatably mounting the reduction gear in reduction gear bearings which are contiguously formed with a load sheave bearing which is formed in the center of the frame. Due to such a constitution, the distance between

the axes of the load gear and the reduction gears which are meshed with the load gear can be made small and hence, the respective gears can be made compact whereby an advantageous effects that the device can be made
5 compact and light-weighted and the number of parts and the man-hours for machining and assembling can be reduced can be obtained.

The invention described in claim 4 is characterized in that a frame for rotatably supporting the load sheave
10 and the reduction gear is comprised of a first and second frames which are arranged close to each other, the load sheave is rotatably mounted in a bearing of the first frame and the reduction gears are rotatably mounted in the second frame which is provided with reduction gear
15 bearings which are contiguously formed with the load sheave bearing. Since the load sheave bearing and the reduction gear bearing can be machined separately, an advantageous effect that the machining of these parts can be facilitated is obtained.

20 Embodiments of the present invention are explained hereinafter in conjunction with drawings.

(Embodiment 1)

The supporting structure of reduction gears and the structure of a load gear in a hoisting/pulling device of
25 the present invention are explained in conjunction with Fig. 1 and Fig. 2.

In the drawings, numeral 1 indicates a frame which is provided with a load sheave bearing and a reduction gear bearing, numeral 2 indicates a load sheave which is
30 rotatably mounted between the frame 1 and a frame which faces the frame 1 in an opposed manner, numeral 3 indicates a load gear which has a diameter equal to or smaller than an outer diameter of a bearing portion of

the load sheave 2 and is integrally formed with the load sheave, numeral 4 indicates a pinion gear, numeral 5a indicates reduction gears a which are meshed with the pinion gear 4, and numeral 5b indicates reduction gears b which are rotatably mounted coaxially with the reduction gears 5a. The reduction gears 5a and the reduction gears 5b form a pair of reduction gears 5 and these gears are respectively meshed with the pinion gear and the load gear. With the provision of a plurality of reduction gears, the transmission torque can be dispersed and the reaction forces in the radial direction between gears which are meshed with each other can be cancelled so that respective gears can be miniaturized and respective bearings can be miniaturized. Numeral 7 indicates the bearing for the load sheave which is formed in the center of the frame 1, 8 indicates bearings for shafts of the reduction gears 5 which are contiguously formed at the outside of the load sheave bearing 7, numeral 9 indicates an upper hook mounting hole, numeral 10 indicates stud bolt mounting holes, numeral 11 indicates stud bolts, numeral 12 indicates an upper hook, numeral 13 indicates a chain which is provided with a lower hook at one end thereof, numeral 14 indicates a mechanical brake which is comprised of brake plates, a brake support, a ratchet wheel and a pawl, numeral 15 indicates a pinion shaft which passes through a through hole formed in the center of the load sheave, is rotatably mounted and is provided with the pinion gear at one end thereof and the mechanical brake 15 at the other end thereof. Numeral 16 indicates a manipulation lever and numeral 17 indicates a casing. Bearings for shafts of the pinion gear 4 and the reduction gear 5 are mounted on the casing 17.

The manner of operation of the device is explained. When the manipulation lever 16 is rotated, the rotational force is transmitted to the pinion shaft by way of the mechanical brake 15. Then, the load sheave 2 is rotated
5 by way of the pinion gear 4 which is formed on one end of the pinion shaft, a pair of reduction gears 5 and the load gear 3. Accordingly, a chain which is wrapped around the load sheave 2 is wound around so as to hoist or draw a load using the lower hook provided to one end
10 of the chain.

In the present invention, as shown in Fig. 1 and Fig. 2, by making a diameter of the load gear 3 equal to or smaller than an outer diameter of the bearing portion of the load sheave 2, the load gear 3 can be
15 preliminarily integrally formed with the load sheave 2. Further, the reduction gear 5 can be rotatably mounted in the bearings 8 for the shafts of the reduction gear 5 which are contiguously formed with the load sheave bearing 7 formed in the frame 1. Accordingly, it is
20 unnecessary to provide an distance for ensuring the wall thickness between the bearings as in the case of the conventional device so that the distance between the bearings can be shortened and respective gears can be miniaturized, whereby the hoisting/pulling device can be
25 miniaturized and light-weighted. Further, the load gear can be preliminarily assembled into the load sheave or the load gear and the load sheave can be formed by an integral molding. Further, the pinion shaft having the pinion gear is also preliminarily assembled into the load
30 sheave. Accordingly, at the time of assembling the load sheave into the frame, it is possible to assemble a plurality of these parts as a unit so that the number of parts and the man-hours for machining and assembling can

be reduced whereby the manufacturing cost can be reduced.

[Embodiment 2]

An embodiment 2 of the present invention is explained in conjunction with Fig. 3 and Fig. 4.

5 According to this embodiment, in a hoisting/pulling device provided with a transmission mechanism equal to that of the embodiment 1, the frame is formed of a first frame 1a and a second frame 1b which are arranged close to each other. A bearing 7a which rotatably supports the
10 load sheave 2 is provided to the first frame 1a, while bearings 8 for shafts of reduction gears 5 are formed in the second frame 1b contiguously with a through hole which allows a bearing portion of the load sheave and the load gear pass therethrough. Further, the load sheave 2
15 is rotatably mounted in the first frame, while the reduction gears 5 are rotatably mounted in the second frame 1b. Due to such a constitution, this embodiment performs the operation equal to that of the embodiment 1. Further, since the load sheave bearing and the reduction
20 gear shaft bearing can be separately machined and hence, the advantageous effect that the machining of these bearings can be facilitated compared to a case in which the contiguous bearings are integrally formed is obtained.

25

Industrial Applicability

According to the present invention, the diameter of the load gear is made equal to or smaller than the outer diameter of the load sheave and the reduction gears are
30 rotatably mounted in the bearings which are contiguously formed with the load sheave bearing. Therefore, it is unnecessary to provide the distance between the load sheave bearing and the reduction gear shaft bearing as in

the case of the conventional device so that the distance between the axes of the reduction gear and the axis of the load gear and the pinion gear can be made small. Accordingly, respective gears can be miniaturized.

5 Further, since the load gear and the load sheave are formed by an integral molding, the pinion shaft which is provided with the load sheave and the pinion gear can be preliminarily assembled as a unit so that the device can be miniaturized and light-weighted and the number of

10 parts or the man-hours for machining and assembling can be reduced so that the hoisting/pulling device can be provided at a low cost.

Claims

1. A hoisting/pulling device comprising a pinion gear, reduction gears which are interlocked with the pinion gear, and a load sheave which is driven by way of a load gear which is interlocked with the reduction gears, wherein the load gear has a diameter thereof equal to or smaller than an outer diameter of a bearing portion of the load sheave and the reduction gears are directly meshed with the load gear.

2. A hoisting/pulling device according to claim 1, wherein the load sheave and the load gear are formed by an integral molding.

3. A hoisting/pulling device according to claim 1 or 2, wherein the reduction gears are rotatably mounted in reduction gear bearings which are contiguously formed with a load sheave bearing which is provided at the center of the frame.

4. A hoisting/pulling device according to claim 1 or 2, wherein a frame for rotatably supporting the load sheave and the reduction gears is comprised of a first and second frames which are arranged close to each other, the load sheave is rotatably mounted in a bearing of the first frame and the reduction gears are rotatably mounted in the second frame which is provided with reduction gear bearings which are contiguously formed with the load sheave bearing.

Fig. 1

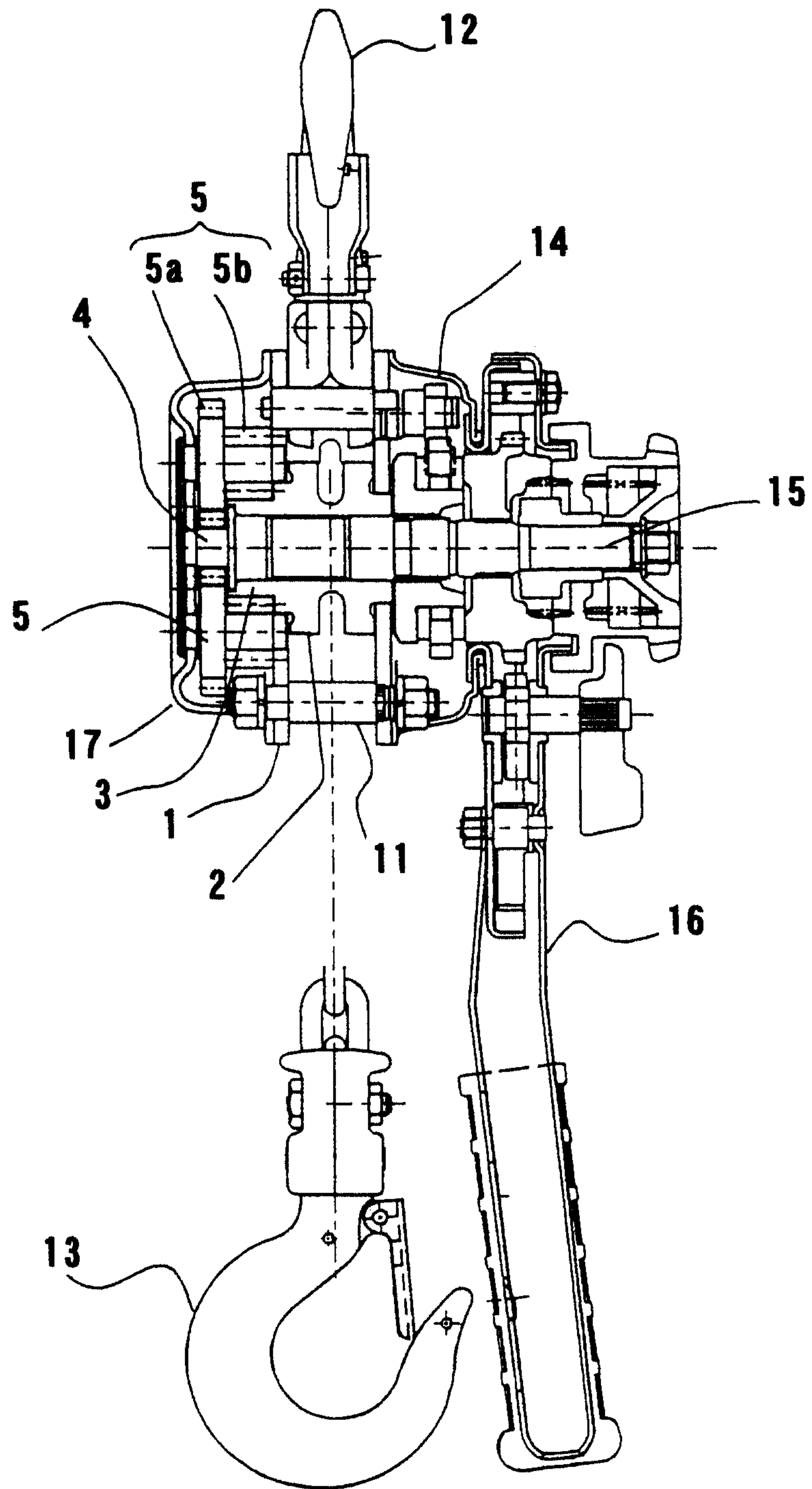


Fig. 2

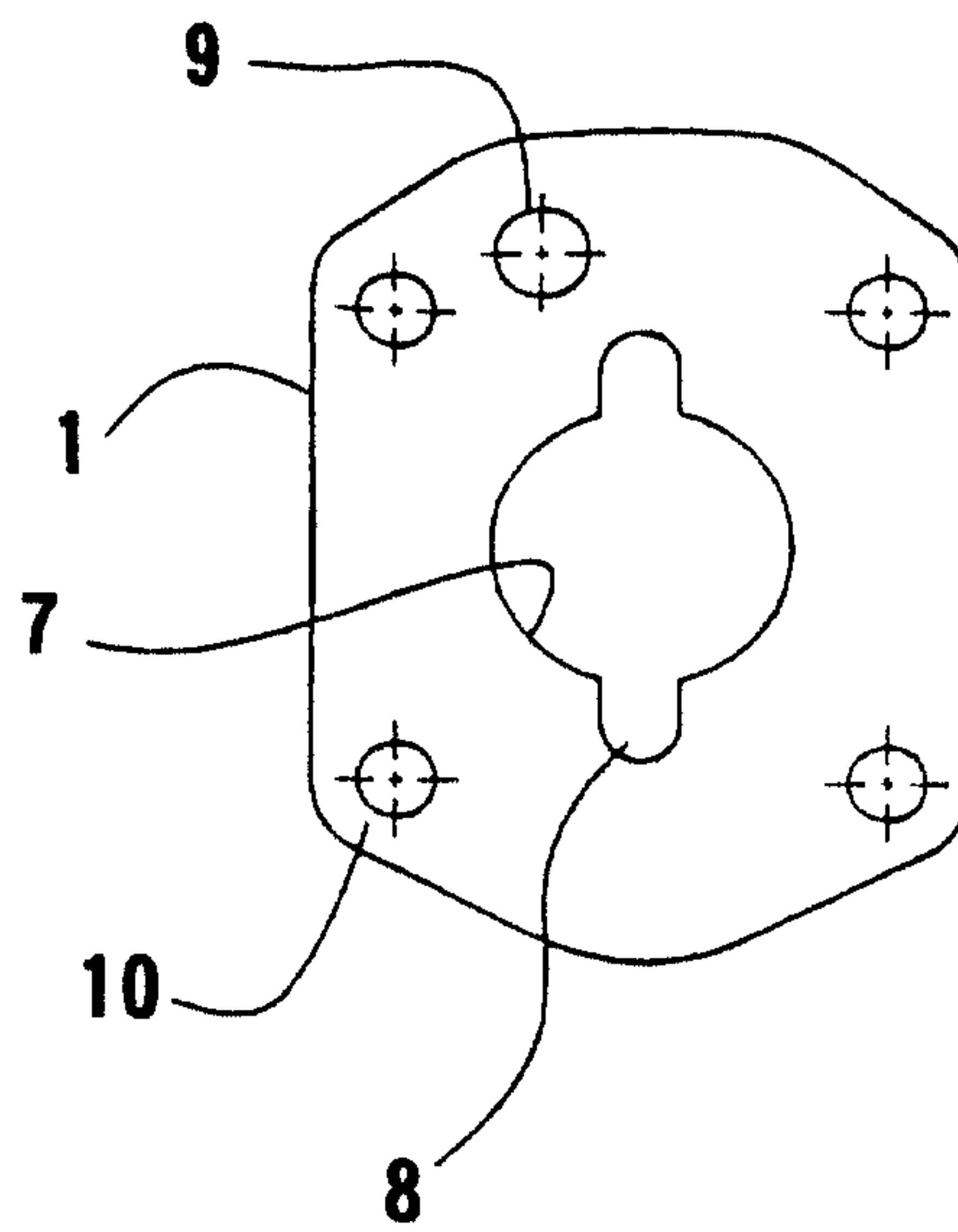


Fig. 3

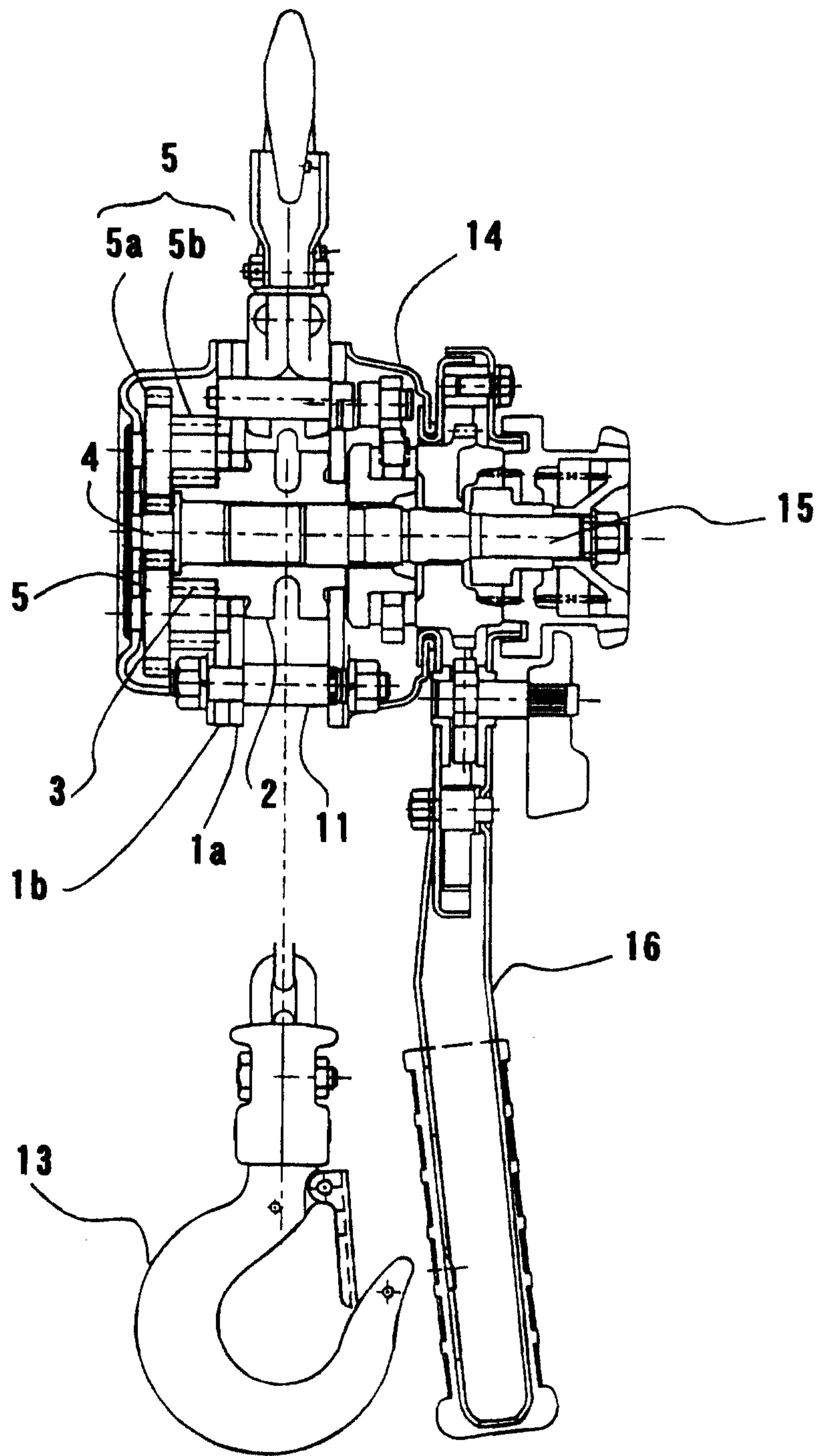


Fig. 4

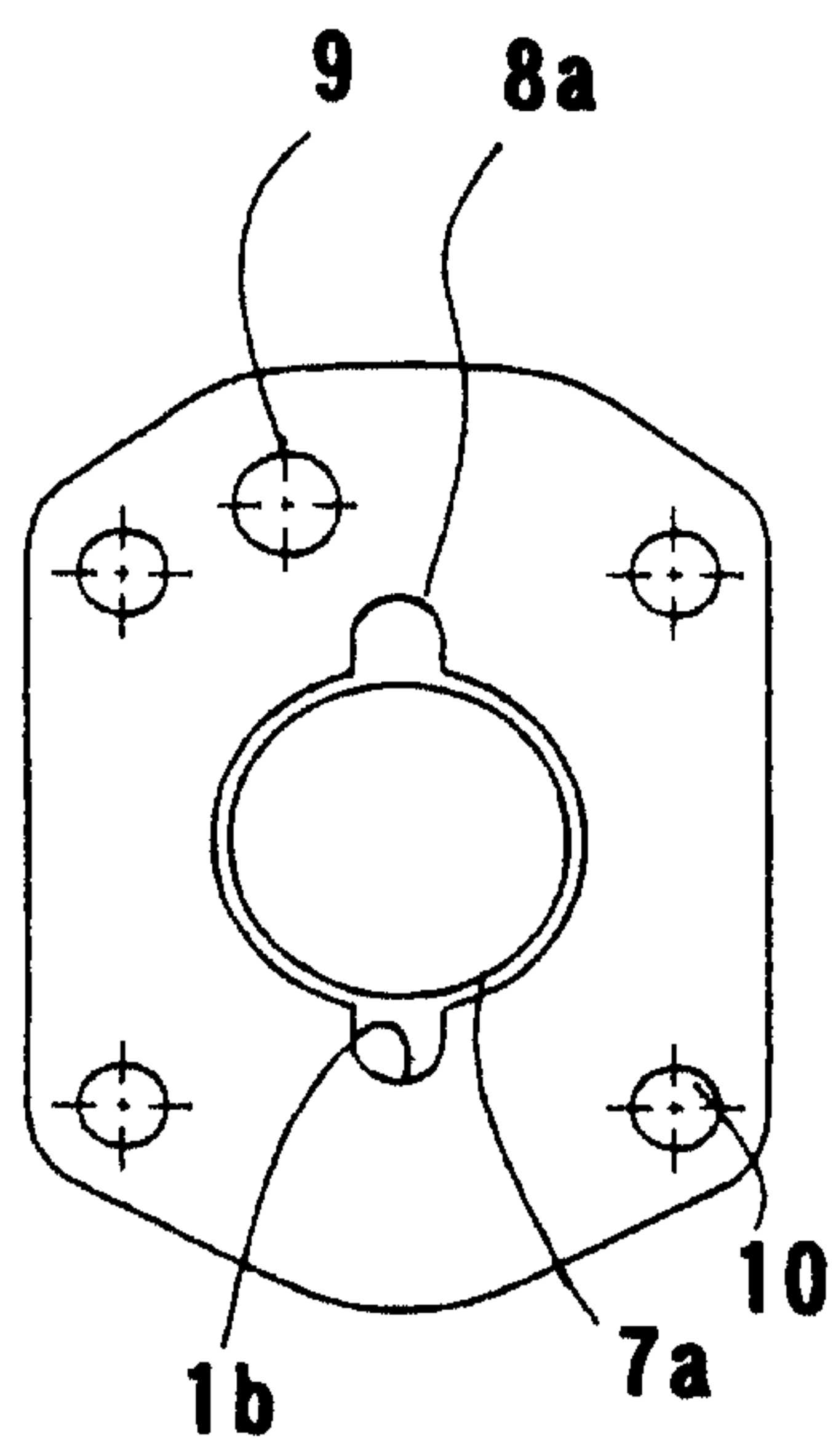


Fig. 5

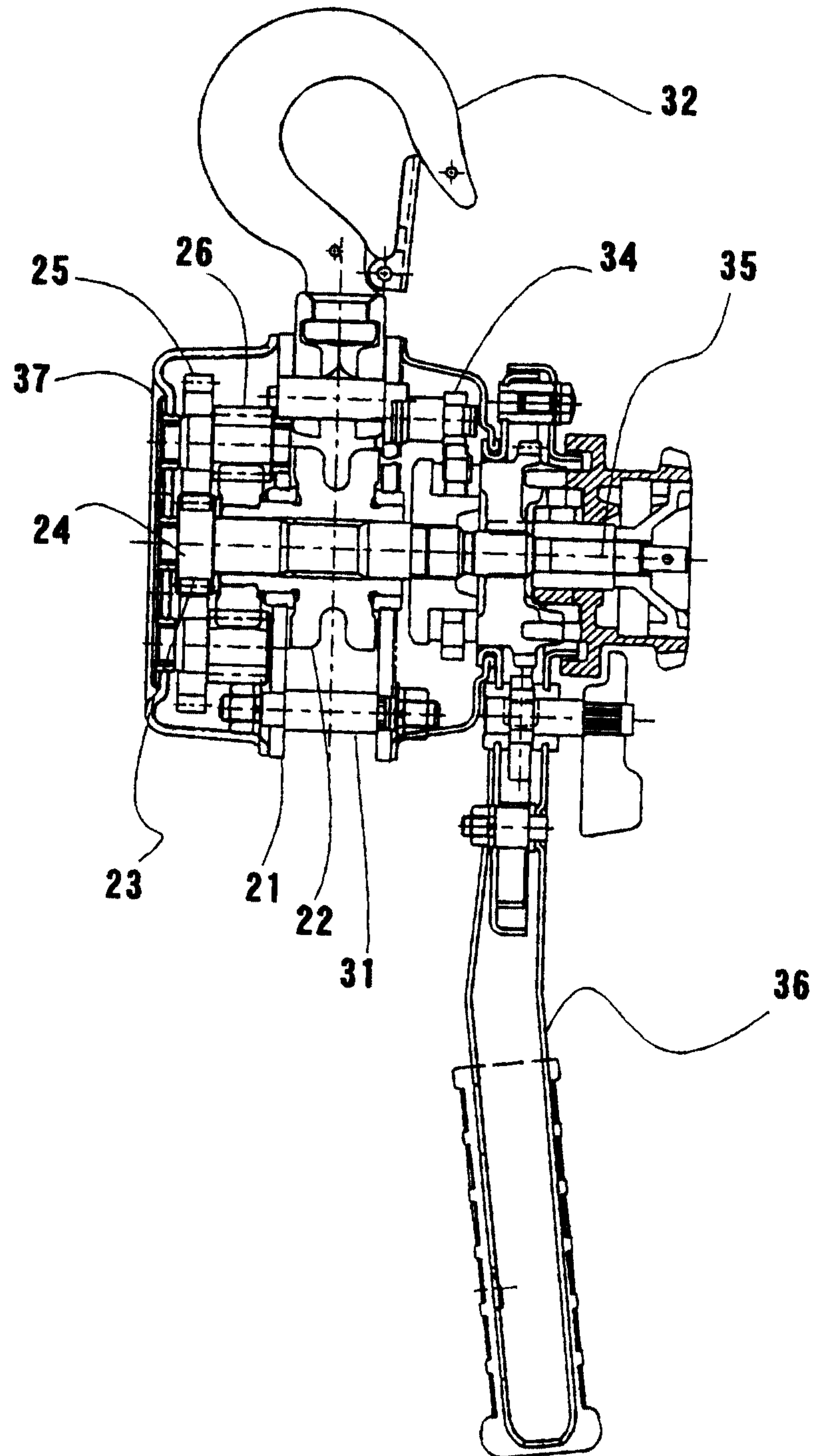


Fig. 6

