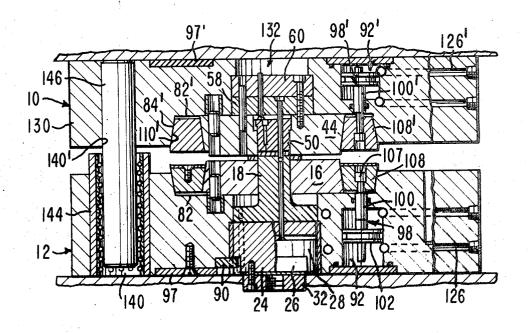
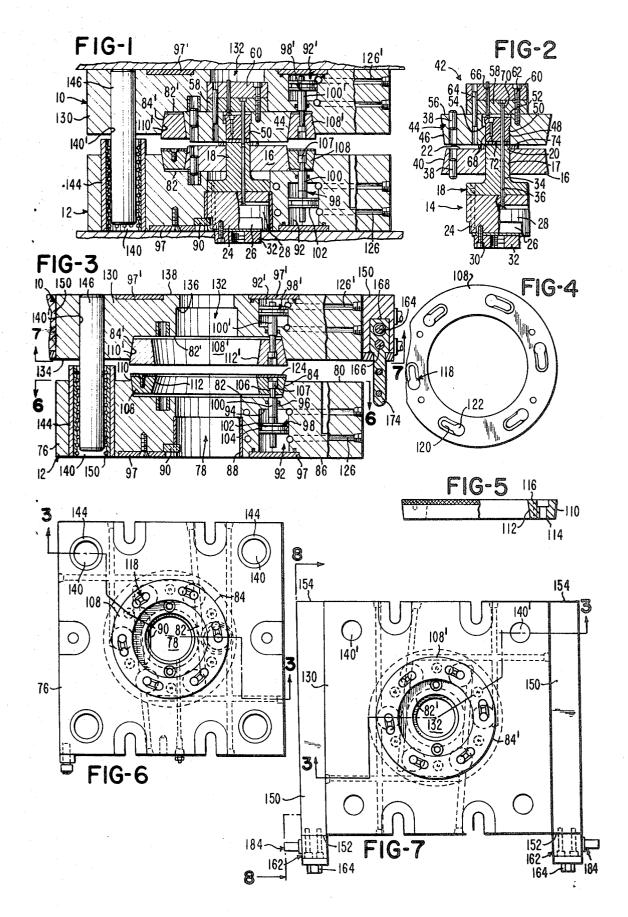
# Waizmann et al.

[45] Aug. 14, 1973

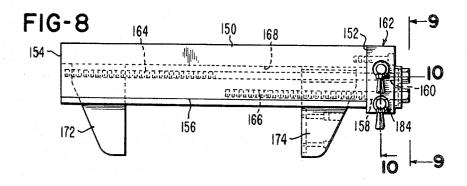
[54]	DIE APPARATUS FOR FINE BLANK STAMPING		2,984,176 3,673,908 3,610,021	5/1961 7/1972 10/1971	Sommer et al	
[75]	Inventors:	George H. Waizmann, Dayton; Robert G. Cook, Sr., New Carlisle, both of Ohio	2,017,247 3,279,297 3,690,209	10/1935 10/1966 9/1972	Hodge	
[73]	Assignee:	The Gem City Engineering Co., Dayton, Ohio	Primary Examiner—Andrew R. Juhasz Assistant Examiner—Leon Gilden Attorney—Jerome P. Bloom			
[22]	Filed:	Oct. 4, 1971				
[21]	Appl. No.	ppl. No.: <b>186,157</b>				
[52]	U.S. Cl	. <b>S. Cl.</b>			ABSTRACT	
[51] [58]	Field of Search 83/698-700, 619, 684-691		Apparatus including a versatile die set particularly advantageous for fine blank stamping, featuring a quick functioning shroud ring for clamping and locating die components with extreme accuracy and precision guides.			
1,715,972 6/1929 Woodward et al 83/685			21 Claims, 11 Drawing Figures			

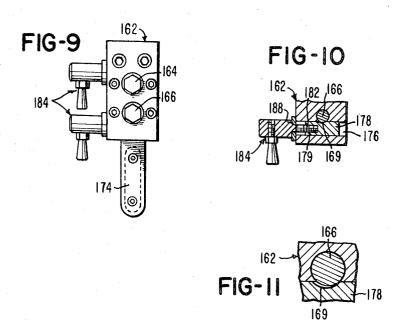


### 2 Sheets-Sheet 1



## 2 Sheets-Sheet 2





#### DIE APPARATUS FOR FINE BLANK STAMPING

#### BACKGROUND OF THE INVENTION

This invention relates to improvements in apparatus and procedures for fine blank stamping and more par- 5 ticularly to devices facilitating the necessary tooling and the maintenance and interchange thereof.

Fine blank stamping is a highly desirable method of producing parts for many articles. It minimizes handling and machining and provides better looking parts. 10 However, it has had minimal application, particularly in the Western Hemisphere. This has been due to several factors including the following:

1. While fine blanking produces parts with relatively smooth cut edges in contrast to the broken jagged 15 edges resulting in use of conventional stamping equipment and procedures, the necessary equipment and tooling therefor is quite costly and the same is highly specialized and limited in application.

2. The means normally employed for tool set up is that involved that it consumes a lot of time, effort and expense. For fine blanking, moreover, the cutting and forming tools in use must be sharpened quite frequently and each time this has heretofore required as much or more time and effort as that required for the original tool set up. As a matter of fact, tool maintenance for fine blanking is two to three times that of conventional stamping operations.

which would facilitate the standardizing and simplifying of the tooling required for fine blanking.

4. An additional time loss and cost factor in conventional fine blanking has been due to the necessity of critical tool spacing and the adjustment of the closed 35 press position on application of each and every tool. This involves several complete press cycles with shut height adjustments until proper penetration of all tools applied has been achieved.

above mentioned problems which exists in the use of conventional fine blanking apparatus and procedures.

#### SUMMARY OF THE INVENTION

acceptance and use of fine blank stamping feasible but it makes it most desirable for many applications. It provides a die set and method of mount thereof lending substantial versatility to available fine blank stamping machines, such as represented by the Osterwalder or 50 Hydrel fine blanking presses. Its application enables, moreover, a substantial reduction in the cost of tool set up and change over in fine blanking operations.

Preferred invention embodiments feature unique locator means for included die components character- 55 ture. ized by power actuated tapered locking shroud rings. Such rings can be quickly and easily applied to their control components and can be readily substituted to meet the needs of a particular installation. Preferred invention embodiments also feature highly utilitarian precision guide means for the materials being worked.

By use of the simple but effective apparatus provided by the invention there can be achieved a saving of as much as 40% in cost of tooling for fine blanking. Moreover, there can be a reduction in cost of production runs, depending on production volume, in a range from 10 to 50%.

Attention is directed to the fact that the invention apparatus and procedures are particularly applicable to the sliding punch type of tooling.

A primary object of the invention is to provide improvements in tooling for fine blanking rendering the same more economical to fabricate, more efficient and satisfactory in use, adaptable to a wide variety of application and unlikely to malfunction.

A further object of the invention is to provide an improved die set for fine blank stamping featuring quick and accurate positioning means for the die components, applied tools, and the materials being worked.

Another object of the invention is to provide improvements in die sets and their application to presses used for fine blank stamping facilitating the positioning of punch and/or die assemblies as units.

An additional object of the invention is to provide improvements in locator means for punch and die units particularly applicable to fine blank stamping facilitating their simple application and substitution.

A further object of the invention is to provide die sets and locator means therefor, and the materials worked thereby, particularly applicable to fine blank stamping presses and the like, possessing the advantageous structural features, the inherent meritorious characteristics and the means and mode of use herein described.

With the above and other incidental objects in view as will more fully appear in the specification the inven-3. Heretofore there has been no means proposed 30 tion intended to be protected by Letters Patent consists of the features of construction, the parts and combinations thereof, and the mode of operation as hereinafter described or illustrated in the accompanying drawings, or their equivalents.

Referring to the accompanying drawing wherein is shown one but obviously not necessarily the only form of embodiment of the invention,

FIG. 1 is a fragmentary view, taken in cross section, illustrating the application of die sets and their locating The present invention is directed to overcoming the 40 means in accordance with the invention in a press applied to fine blank stamping;

> FIG. 2 is a cross sectional view illustrating an operative association of the die sets per se of FIG. 1;

FIG. 3 is a cross sectional view illustrating the details The present invention not only makes a more general 45 of die set locating and clamping apparatus and material guide apparatus in accordance with the invention;

> FIG. 4 is a plan view of a shroud ring featured by the invention, a cross section being shown in FIG. 5;

FIG. 6 is a view taken on line 6-6 of FIG. 3;

FIG. 7 is a view taken on line 7-7 of FIG. 3:

FIG. 8 is a view taken on line 8-8 of FIG. 7;

FIG. 9 is taken on line 9-9 of FIG. 8;

FIG. 10 is taken on line 10-10 of FIG. 8; and

FIG. 11 is a fragmentary detail of the invention struc-

Like parts are indicated by similar character of reference throughout the several views.

With particular reference to the illustrative but not limiting embodiments of the drawings, FIG. 2 thereof shows in detail a cross section of a typical die set in accordance with the present invention. As will be further described, the upper and lower halves of this die set are respectively applied to a die holder 10 and a punch holder 12, the latter of which are in turn secured to the backing plate surfaces afforded by the upper and lower operating elements of a press such as used in fine blank stamping operations.

3

It is to be understood that the elements of the invention embodiment as here disclosed are described with reference to their orientation shown in the drawings. Of course, the included tool elements may be appropriately provided in various manner to suit the particular 5 application.

The lower half 14 of the punch and die assembly illustrated includes a stinger plate 16. Bearing for reciprocation in a central aperture in the plate 16 is the shank 17 of a punch 18. As shown, the operating end 10 20 of the punch is uppermost, being adapted to project through the upper material accommodating surface 22 of the stinger plate 16. At its end remote from the plate surface 22 the punch 18 has an expanded body portion the head 24 of which includes a slot-like cavity 26 for 15 a knock-out plate 28. Secured to the head 24, shown lowermost, by screws 30, is a shank plate 32. The latter is appropriately adapted to provide for connection of the punch operating component embodied in the fine blanking press to which the die set may be applied.

In the case illustrated the punch body includes a vertically orienting through passage 34 opening at one end to the cavity 26 and at its other end from the stinger plate surface 22. The passage 34 accommodates an ejector pin 36 the one end of which bears, under the influence of gravity, on the upper edge of the knock-out plate 28 and the other or upper end of which is operable, under the influence of the knock-out plate, to serve a slug ejecting function.

Press fit in and projecting downwardly from suitably located apertures in the stinger plate 16 are dowel pins 38 which serve a locating function.

Particular attention is directed to the conically tapered peripheral surface 40 of the stinger plate 16. As shown the surface 40 converges in an upward sense. The purpose of this will soon be obvious. The upper half 42 of the die set includes a die plate 44 the outer peripheral surface 46 of which is tapered similarly to the surface 40, but in a reverse sense. Moreover, the plate 44 also has apertures accommodating projected dowel pins 38. However, in this instance the pins project from and perpendicular to the surface of plate 44 remote from the stinger plate 16.

The die plate 44 is provided with a central aperture 48 positioning in use, as shown in FIG. 2, to align directly with the central aperture and the punch in the stinger plate 16. Bearing in and on the wall defining the aperture 48, is a shedder element 50. The head or upper end of the shedder is expanded and lodged for movement within the limits of an expanded upper end portion 52 of the aperture 48. This expansion of the upper end of the aperture 48 produces a shoulder 54 which limits the downward travel of the shedder in a manner believed obvious. The amount of shedder travel so provided is just sufficient to accommodate a part from the material being formed and to provide for its ejection from the die plate. Seated over the aperture 48 and in abutted relation to the uppermost surface 56 on the die plate 44 is a retainer plate 58 and superposed on the latter back up plate 60. The plates 58 and 60 are jointly secured to the die plate 44 by socket head screws 62 and properly positioned in respect to the die plate by insertion of dowel pins 64 press fit in aligned openings thereof.

The shedder 50 is backed by an actuator pin 66 projected through and in bearing relation to the retainer plate 58 and back up plate 60.

4

Merely for purposes of illustration in this instance, in the example of FIG. 2 the shedder 50 is shown to incorporate an oil seal breaking pin 68 and to have extendd therethrough in bearing relation thereto, a further punch 70, the latter having its expanded head based in the retainer plate 58 and capped by the back up plate 60.

As will be obvious, from the illustration of FIG. 2 showing a part 72 in process of being formed from sheet material 74, the main punch 18 serves to determine the outline of the part while secondary punches such as 70 are operated to achieve required cut outs in the part. Of course, the number and character of the punches can be varied to suit the particular application.

FIG. 3 shows details of the holding and locating apparatus for the respective halves of the die set shown in FIG. 2. The lower thereof, the punch holder 12, comprises a relatively thick generally rectangular base plate 76, which is suitably apertured for bolting and/or 20 clamping to the arm portion of a fine blanking press. The plate 76 has a central aperture 78 substantially expanded by countersinking at its uppermost surface 80 to form inwardly thereof a circular recess the base of which is defined by a relatively wide annular shoulder 82. The wall surface 84 which rims this recess is contoured to conically expand from the shoulder 82 to the plate surface 80. The end of the aperture 78 which opens from the lowermost surface 86 of the plate 76 is slightly enlarged in diameter, by counterboring, to nest therein a thin sleeve type bushing 88 the inner diameter of which is slightly less than that of th immediately adjacent portion of aperture 78.

As will be seen in FIG. 1, the bushing 88 accommodates and provides a bearing surface for the periphery of the head end of the punch 18 and its reciprocating element (not shown). The latter of which, provided in connection with the ram in the associated press (not shown), is connected to the punch through the medium of the shank plate 32 in suitable manner well known to a mechanic in the art. The details of such connection are not per se necessary to an understanding of the present invention and are therefore not shown or further described.

The bushing 88, as seen in FIG. 3, is provided with a slot in its lower edge through which is thrust a key 90 anchoring in a mating recess in the punch head 24 to radially locate the punch assembly.

A series of cylindrically shaped recesses 92 are formed in the bottom surface 86 of the plate 76 on a circle concentric with its central aperture 78. The uppermost or base surface 94 of each cylindrical recess 92 has formed therein a central aperture providing one end of a coaxial passage 96, the other end of which opens from the shoulder 82. At their lowermost open ends the recesses 92 are capped by an annular platelike insert 97 which seats in nested relation in an accommodating recess in the plate surface 86.

A piston unit 98 is housed in each cylindrical recess 92. This unit includes a piston rod 100 mounting adjacent one end thereof an expanded plate like head 102 the periphery of which is circumferentially recessed to accommodate the inner peripheral portion of a projected resilient ring seal 104. As will be seen from FIG. 3, the periphery of the head 102 and the peripherally projected seal 104 will bear for movement on the surrounding recess wall. The seal 104 insures a division of the recess in which the piston unit is accommodated

into two chambers, one being defined between the piston head and the cap plate 98 and the other between the piston head and the base 94 of the pertaining recess. The piston rod 100 is shown to project upwardly through the passage 96 and to bear on the wall thereof 5 and extend outwardly beyond the shoudler 82. The extension of the rod 100, shown uppermost in FIG. 3, is reduced in diameter in the portion which projects upwardly immediately of the shoulder 82 and is thereby formed to include a necked portion 106 the upper pro- 10 jected extremity of which terminates in a relatively expanded disc shaped knob 107. As will be further described, the upward and outwardly projected ends of the rods 100 mutually serve as a means for a quick release mounting of a shroud type locking ring 108.

As seen in FIGS. 4 and 5, the ring 108 has an annular form. Both its outer peripheral surface 110 and its inner peripheral surface 112 are formed as segments of a conical surface. The shape of the surfaces 110 and formly from a narrow base 114 of the ring 108 to its opposite surface 116 which is relatively expanded in a radial sense. It is noted that the surface 110 is formed on the same angle as the surface 84 and the surface 112 is formed on an angle ocmplementary to that of the 25 stinger surface 40. Formed in the ring 108, of a uniform radius, is a circularly spaced series of arcuate slots 118, opening from and perpendicular to the surfaces 114 and 116. At the upper ends thereof, which open from the surface 116, the slots are of uniform width in a ra-  $^{30}$ dial sense. However, at a level spaced immediately below the surface 116 the major extent of the length of each slot is reduced in width by an inward projected wall segment forming a U-shaped shoulder 120 recessed immediately below the surface 116. In this man-  $^{35}$ ner the slots 118 are so provided as to have the major extent thereof a width which in this instance is designed to accommodate a necked portion 106 of a piston rod 100 while one end 122 thereof is open sufficiently to accommodate the projection therethrough of the disc 40 shaped knob 107 on the upper end of the applied rod.

As will be readily seen, the slots 118 provide a means whereby the shroud ring 108 may be readily applied, removed, and replaced. The ring may be positioned to dispose slot portions 122 in respective alignment with 45 the heads 107 of the rods 100. On positioning the ring thusly and on dropping the same over the knobs 107, the ring may be freely applied to seat to the outer periphery of the shoulder 82. Through the medium of a slight and quick rotation of the ring, the ring can then be releasably locked. The necked portions 106 of the rods move into the narrowed portions of the slots 118 and the knobs 107 wedge on the related shoulders 120. In the example illustrated, the surface 116, uppermost on the ring 106, is capped by a suitably secured annular plate 124 of similar configuration. In the manner and with the construction described the ring 108 may be quickly connected for limited movement with the piston units 98, to and from the shoulder 82 and the surrounding wall surface 84.

As schematically illustrated in FIG. 3, hydraulic lines 126 are suitably led to either side of each piston head 102, the lines being incorporated in and forming part of a conventional hydraulic control system whereby on 65 actuation of suitable conventional controls fluid may be moved to and from the chambers to either side of the heads in accordance with the need for selective

projection and retraction of the piston rods in a manner and for purposes deemed obvious. As to the hydraulic control system, the details thereof are well within the skill of a mechanic versed in the hydraulic art and are therefore not further described.

FIG. 3 shows that the die holder unit 10 is almost identical in configuration and composition to the punch holder unit 12, similarly embodying piston units 98' identical with the units 98 and similarly functioning, though in a position inverted from that of the units 98 in the punch holder 12. As shown, the die holder unit 10 comprises a relatively thick generally rectangular plate 130 essentially the same as the plate 76 except for the configuration of certain apertures therein. In 15 further describing the die plate 130 and the related parts, where the elements are identical to those in the punch holder unit, the same numerals will be used but with a "prime" symbol.

The plate 130 has a central aperture 132 the end of 112 is such to cause them to diverge upwardly and uni- 20 which shown lowermost in FIG. 3 is expanded by a counter-sinking process to produce inwardly of the bottom plate surface 134 a recess defined by a shoulder 82' rimmed by a peripheral wall 84' conically convergent as the wall surface 84 but in a reverse sense. The convergence in this instance is directed upwardly to the base 82'. The aperture 132 beyond the shoulder 82' is stepped in diameter to provide its minimal diameter at its upper end, forming thereby, in spaced relation to the shoulder 82' a further shoulder 136. In the case of the plate 130 the cylindrical recesses 92' accommodating the piston units 98' are provided inwardly of its uppermost surface 138, the open ends of the recesses being appropriately capped by an annular plate 97' as described with reference to the punch holder.

Viewing FIGS. 6 and 7 of the drawings, it may be there seen that the plate 130 is provided with four rectangularly positioned apertures 140' adapted to align with relatively larger apertures 140 formed in the plate 76. The latter accommodate, the length thereof and projected upwardly therefrom, sleeve like bushings 144. Projecting in the bushings 144 are guide posts 146 the uppermost ends of which are press fit or otherwise suitably fixed in the apertures 140' in the plate 130. As seen in FIG. 3, bearing means 150 are provided between the sleeve 144 and the portions of the guide posts which move therein. Means are thus provided to correlate the relative and respective movement of the punch holder and the die holder. It will be also obvious from FIG. 3 that the projected ends of the piston rods 100' are adapted to receive and releasably mount a locking ring 108' in a manner as described with reference to the ring 108, the rings being identical in configuration.

It will be obvious, looking to FIGS. 1, 2 and 3 of the drawings, that absent the ring 108' the upper half of the die set shown in FIG. 2 may be inserted as a unit in the die holder. The die plate 44 thereof will seat on the inner peripheral portion of the shoulder 82', the retainer plate 58 and the backup plate 60 nesting in the aperture 132 to have the latter abut the shoulder 136. On application of the ring 108' to the projected ends of the piston rods 100' and retraction of the rods, the ring 108' is pulled inwardly to the shoulder 82' and the wall surface 84' to contain and fix the position of the die plate, the retainer plate and the back up plate, and the contained operating parts in a manner believed clearly evident. The mating nature of surfaces 110' and

84' insures smooth seating of the ring 108' while the sloping surface 112' will face, abut to and pull up on the complementary opposite facing peripheral surface 46 of the applied die plate for a centered and precise seating thereof.

Of course, the respective plates 130 and 76, through the medium of slots and apertures indicated, are suitably secured to opposing plate portions in the fine blanking press to which they are applied in a conventional and obvious manner.

Thus, there is enabled, by the construction described, a simple and quick assembly of the die set to the punch and die holder units. Moreover, the machinery so provided in a fine blanking press can be quickly adjusted so that in operation of the punch and die unit supplied there will be sharp, clean and precise forming of parts. The illustration in FIG. 2, as indicated previously, is merely by way of example and not to be constructed as limiting. The punches employed will depend upon the particular needs of a particular situation.

What is important is that the construction of the punch and die unit as here provided and the means for clamping the same is that simplified as to substantially reduce tooling requirements for fine blank stamping as the respective halves of the die set and the applied tools, for whatever purpose required. It is believed clear that with the use of the invention concept the halves of the die sets may be assembled as units and simply inserted in their holding plates whereupon the  $^{30}$ clamping lock rings 108 and 108' can be slip fit, turned and with a simple hydraulic actuation of the piston units 98 and 98', a firmly and precisely secured and the pertaining tool elements thereof will be precisely positioned. The invention lends itself to accommodating any variety of dimension of die or stinger plates, the locking rings being correspondingly dimensioned at their inner periphery. Moreover, in the use of a clamping device as described, of ring or other shape, the die and stinger plates may be standardized, it being required only that they be adapted to accommodate a number of tools at various positions thereof so to enable that the same plate may be utilized for various functions. Also, there may be a variety of interchangeable units made available which may be interchanged 45 utilizing the same locking rings. In any event the whole lends itself to a highly versatile system for creating an expanded and economical usage of fine blank stamping procedures.

Effectively the invention has provided a unique approach to fine blank stamping in that it facilitates the use of a master die set for a given range of parts, requiring only essential standardized interchangeable components to produce a particular part. This taken together with the unique principle of using simply actuated tapered locking shroud rings to clamp and locate the die components enables the extreme accuracy in set up necessary for fine blank stamping. As mentioned previously, the unobvious but highly beneficial effect is that there results in the use of the invention for fine blank stamping a reduction of approximately 40% in the normally anticipated tooling costs. As to production costs, the savings enabled by the invention will vary in proportion to volume of created parts, the greatest advantages being associated in low volume part runs as the result of drastically reduced set up time enabled by the invention. Those versed in the art will readily compre-

hend this fact. This is extremely important in view of the substantial incidence of low volume part runs required for highly complicated mechanisms. The quick change capabilities enabled by the invention may be readily comprehended in that less than five minutes is normally required for set up. Economy thus achieved can reduce low volume production costs by as much as 50%. Where there is high volume production, the reduction in production costs is still significant since fine 10 blanking requires tool maintenance at two to three times the rate of conventional stamping, each sharpening requiring a new set up. As to achieving more rapid shut height adjustments, the invention system enables standard tool height to be maintained within the tool, not the press. This enables a press adjustment in one closed press position, insuring complete stock penetration in the first press cycle.

Having set forth a primary aspect of the invention, attention is now directed to the improvements created 20 by the direct addition to and integration with the die holder of simple but precision guide means for the material to be worked.

Guide means for the material being worked in the die set of the invention are in accordance with the invenwell as to reduce the time in removing and replacing 25 tion fixedly attached to the material input side and the material output side of the die plate 130 forming part of the die holder 10. In this respect reference is made to FIGS. 3, 7 and 8 of the drawings. As here seen, the stock guide units applied to the opposite sides of the plate 130, which are identical, include an elongate generally rectangular body 150 of a length corresponding to the length of the side edge of the plate to which the same is to be secured. The body 150 includes parallel end faces 152 and 154 perpendicular to a planar bottom surface 156. Fixed in abutted relation to the face 152 of the body 150 is a further rectangular body 162 which is relatively narrow in the sense of the length of the body 150 and forms a continuation thereof. Note in FIG. 8 that the body 162 has a portion which depends below the surface 156 fo the body 150. The body 162 includes vertically spaced apertures 158 and 160. Projected through aperture 160 is an elongate cylindrical screw 164 which extends through an aligned slot 168 formed inwardly of the bottom of the body 150, the length thereof. Screw 164 has its head abutting a washer thereabout interposed between the head and the adjacent surface of the body 162. Similarly a screw 166 is engaged through the aperture 158 in the body 162 to extend within the slot 168 spaced vertically below the screw 164. The head of the screw 166 is disposed and spaced similarly from the body 162 by an appropriate washer type device affording a bearing sur-

Slots 168 opens at its bottom through the surface 156 of body 150 where its opening is reduced by parallel plate elements fixed to the surface 156. The slot 168 accommodates the projection therein of the upper ends of jaw members 172 and 174. The member 172 is at the end of the body remote from the surface 152 and at its upper innermost end has a threaded bore adapted to align with aperture 160 and have projected therethrough the operating extremity of the screw 164. The jaw 174 has in the one case a threaded aperture adapted to align with aperture 158 and to receive therethrough, in threaded engagement therewith, the screw 166. At its uppermost end the jaw 174 has a further aperture accommodating the projection therethrough of the screw 164. The respective screws 164 and 166 are based in their respective accommodating apertures 160 and 158 so, that on rotation thereof, depending on the direction, the jaw 172 may be moved to and from the jaw 174. Likewise, the jaw 174 can be 5 caused to move to and from the jaw 172.

It will be noted that the jaws 172 and 174 are so contained and their most adjacent faces are so shaped and positioned that such faces are precisely parallel in the areas which depend below the surface 156 of the body 10 150. Suitable bearing inserts may be provided in the adjacent material guiding faces of the dependent portions of the jaws 172 and 174 in a manner believed obvious.

As seen in FIGS. 8, 9 and 10, the body 162 has openings 176 from front to rear thereof respectively inter- 15 secting the aperture or passage therein constituting the aperture 160 on the one hand and the aperture 158 on the other. The one opening which is uppermost intersects the upper portion of the passage formed by the aperture 160 and the one lowermost intersects the lower portion of the passage formed by the aperture 158. As seen in FIG. 10, in this lowermost opening is positioned a pinlike element 178 having a cavity 169 defined by a recess in its surface portion which intersects the through pasage defined by the aperture 158. In one end of the pin 178 is a counterbore forming a cylindrical recess 179 the wall of which is threaded to threadedly accommodate a screw extension 182 of a cylindrical body projecting outwardly of the aperture 30 162 and forming part of a control handle 184. The body of the control handle of which the screw 182 forms an extension is relatively enlarged in diameter. This provides a shoulder 188 facing the outer face of the body 162 and there is a washer interposed between 35 this shoulder and the face of the body 162 surrounding the opening 176. The significant aspect of the cavity 169 transversely disposed in the outer surface of the pin 178 is that its base is arcuately formed and on a larger radius than that of the screw 166, at the portion 40 thereof which projects through the body 162. On turning the handle 184 one way or the other, the same being fixed against longitudinal movement in appropriate manner, the connected screw will turn within the pin to move its cavity 169 in reference to the lower por- 45 tion of the screw 166 which nests therein. As may be particularly seen in FIG. 11, when one wishes to lock the screw in a particular position, one need only turn the control handle to move the pin into a position wherein one extremity of the cavity wall wedges against 50 the screw 166. By this means there is a simple and positive lock of the screw once it has been adjusted in an obvious manner to position the threadedly engaged jaw.

Of course, the upper control for locking the screw 164 is similarly formed and applied to an upper peripheral surface of the portion of the screw 164 which passes through the aligned opening of the upper passage or aperture in the body 162.

From the foregoing, it will be seen that there is fixedly integrated to each of the respectively opposite input and output sides of the plate 180 a guide device affording a very simple construction wherein each dependent material guiding jaw may be precisely and individually positioned. Moreover, there is a very simple but effective expedient of locking a screw to prevent its rotation once the position of the jaws are established.

As may be seen, by the physical integration in the plate 130 of the dependent jaws 172 and 174 which may be precision oriented quickly and simply, one provides a means wherein material fed to and between the punch and die elements of the die set will be accurately and precisely guided. This enables optimal positioning of the material to be worked and a maximal usage of the material in creating the desired parts.

While the guide means may be independently employed in other areas, it is particularly significant when incorporated in the stationary plate 130 of the punch portion of the described die set. It enables that a complete and utilitarian unit may be achieved in a highly economical fashion. Moreover, the controls and manipulations provided enable those not versed in the art to readily handle the same. Of utmost importance, the stock guides are universal in application since they may function equally well with interchange of tooling. Changes in tooling do not necessitate specifically designed guides as normally required in each different fine blanking set up.

From the above description it will be apparent that there is thus provided a device of the character described possessing the particular features of advantage before enumerated as desirable, but which obviously is susceptible of modification in its form, proportions, detail construction and arrangement of parts without departing from the principle involved or sacrificing any of its advantages.

While in order to comply with the statute the invention has been described in language more or less specific as to structural features, it is to be understood that the invention is not limited to the specific features shown, but that the means and construction herein disclosed comprise but one of several modes of putting the invention into effect and the invention is therefore claimed in any of its forms or modifications within the legitimate and valid scope of the appended claims.

Having thus described our invention, we claim:

- 1. A die system, particularly advantageous for fine blank stamping, comprising a tool or die holder unit including a plate-like base, said base including a surface portion for application thereto of a tool or die plate, and means positioning peripherally of the applied tool or die plate arranged to releasably mount a device movable to and from a clamping relation to the tool or die plate to fix the same in a desired position on said base, said peripherally positioning means including reciprocable means, said device being relatively rotatively adjustable to utilize movement of said reciprocable means in one sense to apply a pressure seating said tool or die plate to said base.
- w.

  2. A die system as in claim 1 characterized by said reciprocable means being arranged at spaced locations on said base.
  - 3. Apparatus as in claim 2 characterized by said reciprocable means including rod-like elements which are arranged to be extensible and retractable in reference to said base, said rod-like elements having projected end portions providing means for the selective coupling thereto of the clamping device.
  - 4. A die system as in claim 3 characterized in that said clamping device has a central opening and constitutes means for framing the applied tool or die plate, said clamping device having portions formed to be releasably engaged by said projected end portions of said

rod-like elements upon relative rotary adjustment of said device.

- 5. A die system, particularly advantageous for fine blank stamping, comprising a tool or die holder unit including a plate-like base, said base including a surface 5 portion for application thereto of a tool or die plate, rod-like elements positioning peripherally of the applied tool or die plate at spaced locations on said base and arranged to releasably mount a device movable to and from a clamping relation to the tool or die plate to 10 forms part of an integrated unit including tool elements fix the same in a desired position on said base, said rodlike elements being arranged to be extensible and retractable in reference to said base and having projected end portions providing means for the selective coupling thereto of the clamping device, said clamping device 15 having a central opening and constituting means for framing the applied tool or die plate, said clamping device and said projected end portions of said rod-like elements having means for interlock thereof by a rotating twist of one relative the other.
- 6. A die system as in claim 5 characterized by said clamping device having a ring form and a tool or die plate applied to said base surface having a circularly outlined periphery, and peripheral portions of each of said clamping device and said tool or die plate being 25 complementary in configuration whereby, on retraction of said rod-like elements, said ring will firmly contain and center said tool or die plate.
- 7. A die system as in claim 6 characterized by said rod-like elements forming parts of piston means em- 30 bodied in said base.
- 8. A die system as in claim 6 characterized by said surfaces of complementary configuration being sloped in opposite senses to provide for one to be contained by the other.
- 9. A die system as in claim 8 in which said base surface and said clamping device having surfaces adapting to mate in seating of said clamping device and said mating surface being sloped in the same sense.
- 10. A die system as in claim 9 characterized by said 40 tool or die plate having in connection therewith cutting and/or shedder elements and the said being integrated to apply to said base as a unit whereupon said clamping device may be releasably connected to said rod-like elements and drawn thereby into a clamping, centering, 45 relation to said unit.
- 11. A die system as in claim 1 characterized in that said clamping device has a continuous frame-like configuration and the tool or die plate applied to said base has an outer peripheral configuration complementary 50 having in its work face a circular insert cavity, relato the inner peripheral configuration of said clamping device, said configurations being formed so that, on draw down of one relative the other, said clamping device serves a centering function.
- base surface portion defining a recess the peripheral wall portion of which has an angular configuration and the outer periphery of said clamping device is similarly angular in configuration whereby on draw down of said clamping device a centering function is produced 60

thereon.

- 13. A die system as in claim 12 wherein a die plate forms part of an integrated die unit including a tool holder, retainer and back up plate, and said base being formed for slip fit insert thereof whereupon the said clamping device may be releasably mounted to and positioned in a centering clamping relation to said die plate as applied to said base.
- 14. A die system as in claim 12 wherein a tool plate and said base is formed for slip fit in said recess of the integrated unit, and said clamping device is formed to slip fit to said reciprocable means and to be moved thereby to a clamping relation to said tool plate.
- 15. In a die set including upper and lower holder halves, one of which serves as a tool holder and the other as a die holder, having work faces to dispose on opposite sides of work means fed therebetween, at least one of said holder halves having in its work face a cir-20 cular insert cavity, insert means having a complementary circular periphery, said insert means being applicable in said cavity, reciprocable means having a sliding mount in said one holder half and movable in a fixed path therein, said reciprocable means and said insert means providing means for interlock thereof to utilize motion of said reciprocable means for fixing and centering said insert means in said cavity.
  - 16. Apparatus as set forth in claim 15 characterized by said insert means including a tool or die plate.
  - 17. Apparatus as set forth in claim 15 characterized in that said reciprocable means is fluid actuable.
- 18. Apparatus as set forth in claim 16 characterized by said tool or die plate being formed with a tapered peripheral edge portion and said portion of said insert means utilizing motion of said reciprocable means being complementary in configuration and arranged to clamp thereto.
  - 19. Apparatus as set forth in claim 15 characterized by said insert means including plural elements each of which has a circular peripheral surface which bears upon the other to achieve an automatic centering of said insert means in the seating thereof in said cavity.
  - 20. A die system according to claim 6 characterized by means on said rod-like elements utilizing an extending motion thereof to apply a lifting force to said ring.
- 21. In a die set including upper and lower halves having work faces to dispose on opposite sides of work means fed therebetween, at least one of said halves tively rotatable insert means having a complementary circular configuration received in said cavity with freedom of axial withdrawing motion, means for locating said insert means in respect to said cavity in a rotary 12. A die system as in claim 11 characterized by said 55 sense to position the same for lacking in respect to reciprocable means slidably mounted in connection with said one holder half and operable for releasably locking said insert means in said cavity against axial withdrawal.

# UNITED STATES PATENT OFFICE CERTIFICATE OF CORRECTION

Patent No. 3,752,028 Dated August 14, 1973

Inventor(s) George H. Waizmann - Robert G. Cook, Sr.

It is certified that error appears in the above-identified patent and that said Letters Patent are hereby corrected as shown below:

Col. 4, line 3, "extendd" is amended to read -- extended --;

line 20, "arm" is amended to read -- ram --;

line 31, "th" is amended to read -- the --.

Col. 5, line 6, "shouldler" is amended to read -- shoulder --;

line 25, "ocmplementary" is amended to read
-- complementary --;

line 26, "of" is amended to read -- on --;

Col. 7, line 18, "constructed" is amended to read -- construed -

Col. 8, line 40, "fo" is amended to read -- of --.

Col. 9, line 61, "180" is amended to read -- 130 --.

Claim 9, line 4, "surface" is amended to read -- surfaces --.

Claim 21, line 9 thereof, "lacking" is amended to read -- locking --.

Claim 10, line 3 "said" should read -- same --.

Signed and sealed this 30th day of April 1974.

(SEAL) Attest:

EDWARD M.FLETCHER, JR. Attesting Officer

C. MARSHALL DANN Commissioner of Patents