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European Patent Office
Office européen des brevets



(11)

EP 1 634 834 A2

(12)

EUROPEAN PATENT APPLICATION

(43) Date of publication:
15.03.2006 Bulletin 2006/11

(51) Int Cl.:
B65H 29/12 (2006.01)

(21) Application number: **05019710.2**

(22) Date of filing: **09.09.2005**

(84) Designated Contracting States:
**AT BE BG CH CY CZ DE DK EE ES FI FR GB GR
HU IE IS IT LI LT LU LV MC NL PL PT RO SE SI
SK TR**
Designated Extension States:
AL BA HR MK YU

(30) Priority: **09.09.2004 GB 0420043**

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(54) Speed control for sheet handling apparatus

(57) A sheet handling apparatus (900) is disclosed having a first sheet feeding means (104) for advancing a sheet along a sheet path, a second sheet feeding means (351) for feeding the sheet along a continuation of the sheet path, driving means (901, 903, 905) for driving the first sheet feeding means independently of the second sheet feeding means and for common drive of

the first and second sheet feeding means, and a drive controller (907) arranged to control said driving means (901, 903, 905) to enable a sheet to be driven along the sheet path from the first sheet feeding means (104) to the second sheet feeding means (351) whilst the second sheet feeding means (351) remains undriven and to enable continued feeding of the sheet by both sheet feeding means.

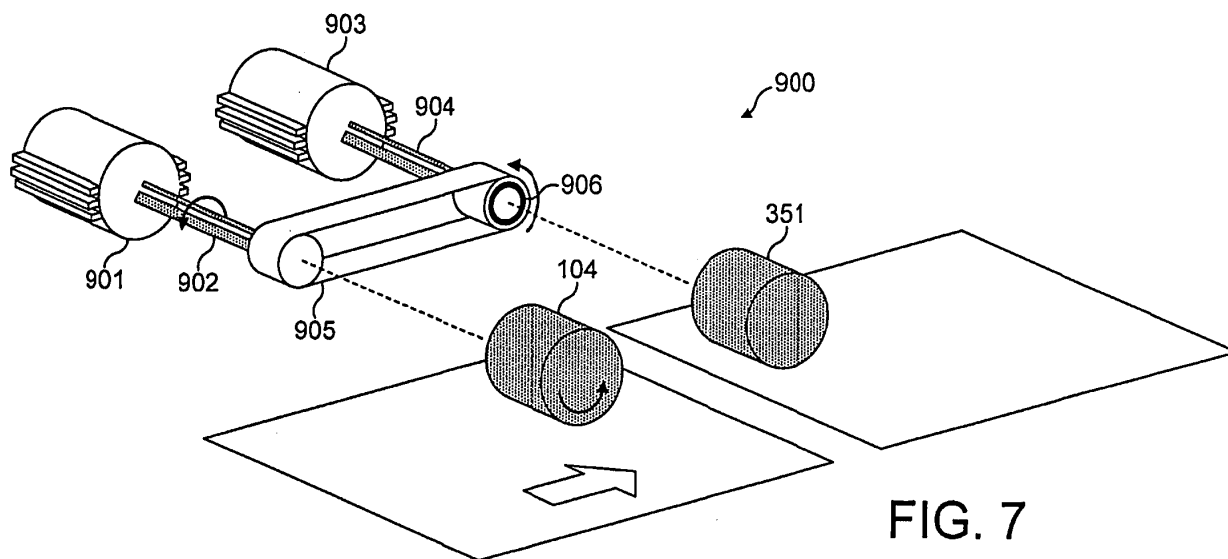


FIG. 7

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Description

[0001] This invention relates to sheet handling apparatus and is applicable to an apparatus and method for processing of elongate elements or articles, and in particular to an apparatus and method for selectively performing a plurality of operations on each of a number of different sheet or booklet elements, as well as envelopes.

[0002] It is well known to provide a machine for successively performing several operations on various sheet elements. For example, operations on an envelope might include flapping, inserting, moistening and sealing, whilst operations on one or more sheets might include collating, folding and inserting into an envelope. It is further known to provide a machine which collates several sheets of paper into a bundle, folds the bundle, places an insert, such as a leaflet or booklet into the bundle, provides an envelope which is held open, inserts the folded sheets into the envelope, moistens the envelope and seals it, before ejecting the envelope into a receiving tray or bin. Each of these operations is distinct and requires a separate and unique processing region within the machine in order to successfully and repeatably carry out the required operation on the respective element. As a result, folder/inserters machines of the type described hereinbefore are typically large and complicated to program.

[0003] Recently, there have been moves towards reducing the size of such folder inserter machines in order to make them more accessible to smaller businesses, such as SOHO (small office/home office) operations. In order to be successful in this environment, the folder/ inserter must occupy a small footprint (i.e. the area of floor/desk-surface occupied), perform reliably, and be easy to control without requiring specialist training.

[0004] GB-A-2380157 discloses a small office folder/ inserter having two trays, and for storing sheets to be folded and the other for storing inserts to be inserted into the sheets. One location is specified for folding said sheets, another location for placing the insert into the folded sheets, and a further location for inserting the folded bundle into an envelope. The machine further comprises a location for storing envelopes, means for opening said envelopes and holding the envelopes open to receive the folded bundle at the inserting location, a section for moistening the flap of the envelope and a section for closing the flap of the envelope to seal it and ejecting the envelope to a receiving tray. Because of the small size and compactness of the machine, it is suitable for performing only a limited number of cycles in a given time period, i.e. it does not have a very high-volume throughput. Further, such machines can lack versatility, since they are suitable only for performing the respective feeding, folding, inserting, envelope opening, envelope moistening and sealing operations on a limited range of sizes of sheets/inserts.

[0005] Large organisations, such as banks, telephone companies, supermarket chains and the government, for example, are often required to produce extremely large

throughputs of specifically-addressed mail to a regional or national audience. Machines capable of producing the high volumes required, whilst simultaneously accurately ensuring that the correct content is sent to the individual recipients, are typically very large, often occupying an entire warehouse. By contrast, existing small office equipment is typically capable of producing mailshots for a few hundred to one or two thousand addressees.

[0006] Demand, therefore, exists for a machine of intermediate production capacity, typically for small to regional businesses, which does not occupy a vast quantity of the available office space. Particularly in large cities, office space is charged at premium rates for each square metre. As such, the cost of running and maintaining a folder/ inserter will also comprise the cost of renting the office space which it occupies.

[0007] For folder/ inserter apparatuses intended for small and medium sized businesses, it is at least desirable, if not necessary, for the machine to be able to accommodate a range of different materials. For example, it will be necessary to accommodate different thicknesses of sheet element, as well as different sizes and numbers thereof. Similarly, any materials to be inserted within a folded package might range from a compliments slip to an entire booklet, including inserts of unconventional size or shape. It is also advantageous for such machines to be able to accommodate different sizes of envelopes, such as A4 and A5, depending on the material to be inserted thereinto.

[0008] When accumulating a document in a mail creation machine it is desirable for the delivery rollers and the receiving rollers/belt to be very closely matched in speed. This prevents the paper being stretched if the feed rolls are too slow or crumpled if too fast. Driving with two independent motors requires very accurate control systems, so the usual method of matching the speed is to mechanically link the two and have one motor for both. This however makes it difficult to stop one sheet in the accumulator and wait for the next to arrive. This invention links the two drives via an overrun clutch allowing the delivery motor to feed independently of the receiving motor whilst maintaining accurate speed matching when the receiving motor is on.

[0009] According to one aspect of the present invention, there is provided a sheet handling apparatus comprising: a first sheet feeding means for advancing a sheet along a sheet path; a second sheet feeding means for feeding the sheet along a continuation of the sheet path; means for driving the first sheet feeding means independently of the second sheet feeding means and for common drive of the first and second sheet feeding means; and a drive controller arranged to control said driving means to enable a sheet to be driven along the sheet path from the first sheet feeding means to the second sheet feeding means whilst the second sheet feeding means remains undriven and to enable continued feeding of the sheet by both sheet feeding means.

[0010] According to a second aspect of the invention,

there is provided a method of feeding sheets in which a sheet is fed by first feeding means to a second feeding means whilst the second driving means remains undriven and the sheet is further advanced by both the first and second feeding means set at a specific ratio therebetween.

[0011] For a better understanding of the invention and to show how the same may be carried into effect, reference will now be made, by way of example, to the accompanying drawings, in which:

Figure 1 is a cross-sectional view of a sheet handling apparatus detailing the different machine sections; Figure 2 is a cross-sectional view of a sheet feeder deskew mechanism;

Figure 3 is a cross-sectional view showing the sheet feeder collation section;

Figure 4 is a schematic view of an accumulator according to an embodiment of the present invention;

Figure 5 is a cross-sectional view showing the accumulator installed in a sheet handling apparatus;

Figure 6 is a cross-sectional view showing the sheet folding section; and

Figure 7 shows a speed compensation mechanism according to one embodiment of the invention.

[0012] Throughout the drawings, like numerals are used to identify like components.

[0013] Figure 1 shows a folder/inserter apparatus 1000 embodying the present invention. This embodiment is exemplary only, and is used to highlight and explain the inventive concept defined by the appended claims.

[0014] Figure 1 shows a cross-sectional view of the folder/inserter apparatus 1000 and schematically shows various sections of the machine. The folder/inserter apparatus 1000 comprises a sheet feeder section including sheet feeders 1, 2, 3 and 4, from which sheets are fed into a collation section 100 where they are collated into an ordered paper stream. The paper stream is then fed along a sheet feed path which merges with an inlet from a convenience feeder 200, which acts as an alternative sheet feeder for certain documents. The sheets then pass through an accumulator section 300 where they are grouped together as an ordered and aligned package. From the accumulator, the sheets pass through a sheet folder 500. Inserts fed from insert feeders 401 and 402 are collated in an insert feeder collation section 450 and then fed into a folded collation. An envelope is fed from an envelope feeder 600 along an envelope transport path 650 to a flapper 700 where the envelope flap is opened and the mouth of the envelope held open at insertion section 750 to receive the folded collation. The collation is inserted into the envelope and the envelope is fed into a final section 800 where the gum on the envelope flap is moistened and the envelope sealed. The sealed envelope is then ejected into a receiving tray or bin.

[0015] Referring now to Fig. 1 in more detail, there is shown an inlet section which includes four sheet feeders

1, 2, 3 and 4. Each of these sheet feeders comprises a respective sheet feeder tray 5,6,7 or 8 into which a stack of sheets may be placed. The sheets in each tray are fed individually into a sheet feed path by respective sheet deskew mechanisms 50 which each act to separate a single sheet from the top of a stack of sheets in the associated sheet feeder tray and to feed the separated sheet into and along the sheet feed path. Four deskew mechanisms 50 are shown in Figure 1, only one of which is identified by reference numeral 50 in Figure 1. The other three deskew mechanisms are either identical or equivalent to that labelled 50. Each of the sheet feeder deskew mechanisms feeds into a common sheet feed path via respective sheet feeding inlet paths P1, P2, P3 and P4. The convenience feeder 200 similarly feeds into the common sheet feed path. All inlets to the sheet feed path from the four sheet feeders and from the convenience feeder 200 merge by a point T within the sheet feeder collation section 100. From the point T, the sheet feed path continues as a single sheet feed path up to the folder station 500. The sheet feed path passes first through the accumulator 300, where a plurality of sheets may be brought together to form an aligned and ordered package. The sheet feed path then passes through the sheet folding section 500 which produces a desired fold pattern in the accumulated document. As shown on the righthand side of Figure 1, a pair of insert feeders 401, 402 are provided. Each insert feeder 401, 402 has a respective feeder tray 411, 412 which holds a plurality of inserts to be inserted into the folded collation. Each insert feeder further has an associated feeder device 400 for feeding a single insert into the insert collation section 450. Inserts fed into the insert collation section 450 are collated together and then inserted into the main folded collation. On the lefthand side of Figure 1 below the sheet feeders 1,2,3 and 4 is located the envelope feeder 600. Envelope feeder 600 holds a plurality of envelopes which are fed along the envelope transport path 650 and into the flapper mechanism 700. The flapper mechanism 700 opens the flap of each envelope and uses mechanical fingers to hold the mouth of the envelope apart at insertion section 750 in order to allow the folded sheets (and any inserts) to be projected into the envelope. The envelope, with inserted documents, then continues along the sheet feed path to the final section 800 in which the gum on the envelope flap is moistened and the flap is sealed. The sealed envelope is then ejected from the folder/inserter apparatus 1000.

[0016] The operation of the folder/inserter apparatus is now considered in more detail with reference to Figures 2 to 6.

[0017] Referring now to Figure 2, the sheet feeder deskew mechanism 50 comprises a separator roller 51 which applies a driving force to the uppermost sheet in a stack in the sheet feeder tray. The separator roller 51 presses against a separator pad 52, normally in the form

of a separator stone. This separator stone 52 prevents more than one sheet at a time from being fed into the sheet feed path by the roller 51. The single sheet removed from the sheet feeder tray by the separator roller 51 is then driven towards a deskew roller pair 53 which is maintained stationary. As the sheet engages the nip defined by the deskew roller pair 53 it is caused to buckle (as illustrated at ). This forces the lead edge of the sheet to align with the nip of the deskew roller pair 53. The separator roller 51 is then stopped and the deskew roller pair 53 operated to drive the sheet along the sheet feed path and into the sheet feeder collation section 100.

[0018] With reference to Figure 3, each sheet fed from a sheet feeder 1,2,3 or 4 or convenience feeder is received in the respective sheet feeding inlet path P1,P2, P3, P4 or P5 defined by guides G1 and G4 to G10. The sheet feeding inlet paths merge into a single sheet feed path in the sheet feeder collation section 100. Sheet feeding roller pairs 101,102,103 and 104 are located along the sheet feed path for forcing the sheets along the sheet feed path.

[0019] In a typical sheet folding/inserting operation involving a four-page document, referring also to Figure 1, the first sheet feeder tray 5 receives a stack of sheets corresponding to page 1 of the document, the second sheet feeder tray 6 receives a stack of sheets corresponding to page 2 of the document, the third sheet feeder tray 7 receives a stack of sheets corresponding to the page 3 of the document, whilst the fourth sheet feeder tray 8 receives a stack of sheets corresponding to page 4 of the document. A single sheet is then fed sequentially from each of the first to fourth sheet feeders. The first sheet from the first sheet feeder 1 passes into and along the sheet feeding inlet path P1 and partially along the common sheet feed path. A sheet is then fed from the second sheet feeder 2 such that the leading edge of the second sheet partially overlaps the trailing edge of the first fed sheet within the sheet feeder collation section 100. Similarly, the third sheet is fed so that the leading edge of the third sheet partially overlaps the trailing edge of the second sheet, whilst the fourth sheet is fed so that its leading edge partially overlaps the trailing edge of the third sheet. This forms a collation of the sheets along the sheet feed path in the sheet feeder collation section 100. The guides G1 to G10 defining the sheet feed path are configured and arranged to ensure that, as the sheets are sequentially fed into the sheet feed path and carried to overlap as described above, they become correctly collated in the intended order.

[0020] Because the requirement is that the adjacent sheets in the sheet collation only partially overlap at the leading and trailing edges, it is possible to drive the sheet collation along the sheet feed path at high speed without requiring a complex control system to ensure that each of the sheets is correctly aligned with those adjacent to it. This enables a high-volume throughput of mail pack-

ages to be achieved.

[0021] Referring now to Figures 4 and 5, the sheet collation is then driven from the collation section 100 into an accumulation section 300 comprising a vertical accumulator 350. Here, as each sheet arrives in the accumulator 350, it is gripped and forcibly advanced through the accumulator by a pair of traction belts 351 running vertically and mutually parallel on a sled 352. A plurality of spring-biased idler rollers 365 to 369 are provided for each traction belt 351 to apply forces F1 to F5 to maintain the most recently-arrived sheet in contact with the traction belts 351. Each sheet fed into the accumulator 350 arrives at an accumulation chamber 364 defined on one side by a sled guide assembly SG including the sled 352 and the traction belts 351 and on the other side by a fixed guide assembly OG including fixed guide 353 and idler rollers 365 to 367. The accumulation chamber 364 is substantially straight and vertical, such that the collation is accumulated into a vertical stack of sheets. At the bottom of the accumulation chamber 364 is an accumulation gate 354 functioning as a stopping device. Each sheet entering the accumulator 350 is driven downwardly through the accumulation chamber 364 towards the accumulation gate 354 by the traction belts 351 until its leading edge comes into contact with the accumulation gate 354. This causes the sheet leading edge to impinge on the accumulation gate 354 and the sheet to become correctly aligned within the accumulation chamber 364. The sheet is then maintained within the accumulation chamber 364 and rests on the accumulation gate 354, whilst further driving of the traction belts causes slippage between the traction belts 351 and the sheet. Thus, once the first sheet has been stopped by the accumulation gate 354, the second and subsequent sheets are consecutively driven into alignment with the first sheet by the traction belts 351 driving each sheet in turn along the accumulation path and against the accumulation gate 354 to form an ordered collation. When all of the sheets in the collation have been successfully grouped at the accumulation gate 354, the accumulation gate opens to allow the collation to progress out from the accumulation chamber 364 along the continuation of the sheet feed path.

[0022] Referring now to Figure 5, it can be seen that the accumulator comprises the fixed guide assembly OG, and movable sled guide assembly SG. The movable guide assembly SG includes driving means in the form of the pair of traction belts 351. The fixed guide assembly OG includes idler rollers 365 to 367 for pressing the sheets to be accumulated against the traction belt 351 and rollers 361 to 363 for pressing the traction belt against the sheets to be accumulated. The movable guide assembly also includes the sled 352 for assisting guidance of the sheets, or collations of sheets, into an accumulated bundle whilst accommodating a variable thickness of accumulation. These features define a section of sheet feed path which is substantially vertical and acts as the accumulation chamber 364. In the embodiment shown, the

means for driving the sheets downwardly towards the accumulation gate 354 is the pair of traction belts 351, although any suitable system of belts and rollers could be used. The present embodiment has two drive belt assemblies which each consist of one of the traction belts 351, a drive roller 355 and a secondary tension roller 356 which holds the traction belt 351 under tension, along with idler rollers 361, 362, 363 in the sled 352 acting in opposition to the idler rollers 365 to 367 in the fixed guide assembly. The idler rollers 365 to 367 associated with the fixed guide 353 could alternatively take the form of miniature drive belts biased towards the fixed idler rollers 361, 362, 363 in the sled, but preferably sprung idler rollers are biased towards the traction belts. Further, idler rollers 368 and 369 are mounted on a further guide component positioned above guide 353 (see Fig. 5). The idler rollers 365 to 369 may be arranged to apply a force to the sheet which varies along the length of the accumulation chamber 364 and around the drive rollers 355 of the traction belt mechanisms 351. Such a variable traction force over the length of the accumulation chamber, preferably ensuring a larger force towards the bottom of the accumulation chamber, reduces the column strength of a sheet required to enable it to resist the frictional driving forces of the traction belts. In the present embodiment, the varied force is achieved by using sprung idler rollers 365, 366 and 367, each of which is biased towards the traction belts 351 by a different spring force, the spring force being largest for roller 367 and least for roller 365. The downward driving force is resisted at the bottom of the accumulator by the accumulation gate 354, but it is important that the traction forces from the driving means do not cause the individual sheets to buckle or concertina.

[0023] In traditional accumulators, the accumulated collation must be mechanically forced in order to propel it further along the sheet feed path. Because contact can be achieved only with the front and rear sheets at any time, the acceleration given to the accumulated collation must be limited in order to ensure that adjacent sheets do not slide relative to one another, thereby spreading apart the accumulated collation. As a result of the vertical orientation of the accumulation path in the present embodiment, a downward acceleration of 1g (i.e. under gravitational force) can be achieved without mechanical forcing. In addition, using additional forcing methods, a further acceleration of 1g may be imparted to the collation without resulting in the separation of adjacent sheets. Hence, accumulated collations emerging from the accumulator 350 of the present embodiment may be accelerated at roughly 2g without resulting in sliding separation of the sheets. This allows for faster progression of the accumulated collation through the folder/insertor 1000, resulting in a higher-volume throughput of sheet packages.

[0024] Referring again to Figures 4 and 5, the operation of the accumulation section will be described in more detail.

[0025] As already outlined, as the sheet collation en-

ters the accumulation section, the individual sheets are engaged by the pair of accumulator driving belts 351. At the accumulator inlet side, a pair of drive rollers 104 (Fig. 5) feeds the sheet material along the sheet feed path towards the drive belts 351. The drive belts 351 are stopped whilst the drive rollers 390 continue to feed a sheet into the accumulator 350. This allows subsequent sheets arriving after the first to be effectively overlapped with the sheet or sheets already in the accumulator 350 to ensure that they are engaged by driving means 351 and accumulated in the correct order.

[0026] According to the present embodiment, there are three methods by which a document may be fed into and accumulated in the accumulator. The first is as described above, where individual sheets are fed from the separate feed trays 1, 2, 3, 4 (Figure 1), loosely collated in the sheet feeder collation section 100, and then accumulated in the accumulator 350. In this mode, the sheets pass directly into the accumulation chamber in the correct order because they are already partially overlapped. As such, the second and subsequent sheets are always received between the sheet(s) already present in the accumulator and the traction belts 351, so that they are driven downwardly and accumulated against the accumulation gate 354 in the correct order.

[0027] The folder/insertor may also operate in two further modes for folding a mail piece and inserting it into an envelope. According to the second method, pre-stapled sheets, for example a five-page document stapled in one corner, are placed in the convenience tray 200. This document is then fed directly to the accumulation chamber, where no further accumulation is required owing to the sheets being stapled. The document then exits the accumulation chamber and is folded and inserted as normal.

[0028] According to the third method of operation, a plurality of ordered, loose sheets are placed in convenience feeder 200 or one of the sheet feeder trays 5, 6, 7 or 8 (Figure 1). These sheets are fed successively one-at-a-time along the sheet feed path and into the accumulator. However, in this mode, the sheets are not partially overlapped in the paper feed path, and this leads to the risk that the sheets will become incorrectly ordered, or incorrectly fed into the accumulator, leading to mis-collated mail packages or a jam in the folder/insertor machine 1000.

[0029] To overcome this problem, a trail edge deflector 380 is provided (Figures 1, 5). In the third mode, the trail edge deflector acts to lift the trail end of a sheet whose lead end is already in the accumulator 350, to thereby ensure that the subsequent sheet to arrive is fed into the accumulator between the previous sheet and the traction belts 351. The trail edge deflector 380 comprises a roller 381 through which there is a passage 382 suitable for allowing one or a plurality of sheets to pass through the roller. The passage is flared at the inlet and outlet thereof to better accept the introduction of a sheet leading edge, to prevent jamming of the folder/insertor 1000.

[0030] In the first and second modes the sheet(s) or stapled document(s), etc. simply pass through the passage in the deflector and into the accumulator.

[0031] In the third mode of operation, the sheets arriving individually pass part-way through the passage, and the leading edge of the sheet enters the accumulator 350 and is contacted by the traction belts 351 to drive it down against the accumulation gate 354. As the trail edge of each sheet reaches the trail edge deflector, the deflector rotates by 180° (anticlockwise as shown in Figure 5). This forces the trail edge of that sheet upwards and it then lies above the trail edge deflector. The inlet and outlet of the passage 382 through the trail edge deflector have then reversed positions and the subsequent sheet enters the passage through what was previously the outlet. This is possible because the passage has a cross-section with rotational symmetry. The subsequent sheet is then guaranteed to be fed into the accumulator underneath the trail edge that was previously deflected, i.e. between the previous sheet and the traction belts.

[0032] This third mode of operation is particularly useful when, for example, a document has been printed by a laser jet printer and is collated in the correct order, and it is not desired to have to sort the individual pages of the document into the appropriate individual sheet feed trays.

[0033] After leaving the accumulator, the collation passes into the folding section 500 which contains a variable folding apparatus. The operation of such a folding apparatus is known, for example from GB-A-2380157. Brief explanation is given here for a more complete understanding.

[0034] Referring to Figure 6, the folding apparatus comprises four rollers 501, 502, 503 and 504 arranged to form three pairs 510, 520 and 530. The leading edge of the collation passes through the first roller pair 510 and into a buckle chute 511 until it reaches an adjustable stop 512, here constituted as a pinch roller pair 513 which selectively stops the collation based on detection of the leading edge position. At this point, the first roller pair continues to feed the sheet collation, causing it to buckle, and causing the buckled portion to enter the nip between the second roller pair 520. This results in the buckled portion being fed through the second roller pair 520 and forming a fold at the buckle, at a predetermined position. The folded edge then becomes the lead edge of the collation and it is fed through the second roller pair 520 into a second buckle chute 521 until it moves into contact with a second stop 522 (which is preferably a pinch roller pair 523) which halts its movement. The second roller pair 520 continues to feed the trailing edge of the sheet collation therethrough. Again, this causes the collation to buckle, and the second buckle is forced into the nip of the third roller pair 530, resulting in a second fold in the sheet collation at a predetermined point in the region of the second buckle.

[0035] By selectively determining the point at which the sheet collation is halted by the stops 512, 522 at each stage, it is possible to always achieve the folds in the

desired position. Further, by appropriately selecting the distance from the roller pairs at which the collation is halted, the same apparatus can selectively perform either a double fold, a "Z" fold or a "C" fold in the sheet collation. Equally, the sheet collation need only be folded a single time, for example simply folded in half. This single fold is achieved by operation of a half-fold mechanism 550. If a half-fold operation is selected, the half-fold mechanism 550 moves in the direction of arrow A to an interference position where it intercepts and redirects the accumulated collation as it exits the first roller pair 510. The collation is then directed immediately through the second roller pair 520, rather than into the first buckle chute 511. Accordingly, the first fold is never made in the collation at the nip of the second roller pair, and only a single fold is created as the collation is buckled in the second buckle chute 521 and the buckle passes through the third roller pair 530, as normal.

[0036] Referring again to Figure 1, after the final fold is made, one or more inserts may be fed from insert feeders 401 and 402 shown on the right hand side of Figure 1. The present embodiment has two insert feeders 401 and 402, which both feed an insert into and along an insert feed path. One or both inserts are then collated in the insert feeder collation section 450 and the collated inserts are held at insert staging area I whilst the sheets are folded. These collated inserts are then fed into the final fold in the sheet collation and form part of the folded document. Typically, these inserts might be booklets, business reply envelopes, compliment slips, product samples, etc. of varied shape, size, thickness and pliability.

[0037] Below the sheet feeders 1 to 4 is located the envelope feeder 600. This holds a plurality of envelopes in a stack, and has an associated mechanism for removing the single uppermost envelope from the stack and feeding said envelope along the envelope transport path 650. The envelope first undergoes a flapping process in flapper section 700, in which the flap is opened. The envelope is then held in the insertion region 750, where it is stopped. Mechanical fingers engage with and hold open the mouth of the envelope. In this state, the folded mail collation (including inserts) is inserted into the envelope by projecting the mail package towards the open mouth with sufficient velocity that its momentum will force it inside the envelope. This mail piece, comprising the folded mail package within the envelope, then proceeds to the sealing and ejection section 800. In the sealing and ejection section there is a moistening device 820 where the gum seal on the envelope flap is moistened. The envelope is then passed through a sealing/ejection mechanism 840. This performs a process which shuts and seals the moistened flap and ejects the envelope from the folder/insert apparatus 1000 into a receiving tray or bin.

[0038] According to the above description, as the sheet collation enters the accumulation section, the individual sheets are engaged by the accumulator driving means

351. In the present embodiment, this accumulator driving means takes the form of a pair of traction belts. In the paper feed path there are sheet feed path driving means 104 feeding the sheet material along the sheet feed path towards the accumulator driving means 351. Regardless of the form of the sheet material - whether it be a partially overlapped collation (as described above), a plurality of single sheets in succession or a stapled pamphlet - it is necessary to accurately match the "feeding speed" of the sheet feed path driving means 104 with the "receiving speed" of the accumulator driving means 351. Referring to Figure 7, there is shown a speed-matching mechanism 900 for accurately matching the feed rates of the sheet feed path driving means 104 and the accumulator driving belts 351, shown schematically in Fig. 7, under control of a system controller 907. Simple speed-matching can be achieved by mechanical interaction - i.e. via a system of gears or a chain and sprocket connection linking the two driving means and forcing them to produce a matched feed rate. However, in order for the accumulator 350 to function properly, it is necessary to be able to stop the accumulator driving means 351 whilst continuing to drive sheet material into the accumulator with the sheet feed path driving means 104. This is necessary to enable the accumulator driving means 351 to be halted by the drive controller 907 whilst a subsequent sheet is fed into the accumulator in the correct order, thereby creating or increasing an overlap between adjacent sheets.

[0039] The mechanism 900 of the present embodiment concerns two coupled rotational driving systems - one comprising traction belts 351 of the accumulator and the other comprising the drive rollers 104 on the sheet feed path. Each driving system comprises a respective motor, i.e. a feeding motor 901 connected via a rotational shaft 902 to the drive rollers 104 and a receiving motor 903 connected via a second rotational shaft 904 to the drive roller of the accumulator driving means 351. Both motors are controlled by a system controller 907. The rotational shafts of the two motors 901, 903 are linked by a drive belt 905 which rotates with the rotational shaft 902 of the feeding motor 901. The drive belt 905 is coupled to the rotational shaft 904 of the receiving motor 903 via an overrun clutch 906. This allows the feeding motor 901 to turn whilst the receiving motor 903 remains stationary. However, when it is desired to feed a sheet by means of the drive belt 351 of the accumulator, the receiving motor 903 begins to rotate. The motors may be controlled such that the receiving motor 903 will be driven at higher power than the feeding motor 901, thereby ensuring that the feeding motor 901 does not "overtake" the receiving motor 903. The feeding motor 901 may be unpowered in this phase, but it is preferred that it should be powered in order to distribute the load required to drive the sheet material between the two motors. Using the arrangement described, it is possible to hold the accumulator receiving motor 903 stationary whilst the feeding roller 104 drives a sheet into overlapping arrangement with another sheet held static in the accumulator

350. When the accumulator driving motor 903 is activated, and both motors are running simultaneously, the sheets will not stretch or become buckled since the sheet will be fed at precisely the same feed rate by both the sheet feed roller 104 and the accumulator drive belts 351.

[0040] Thus, two independent drive systems are provided, each consisting of: a motor driving a roller set, via a combination of shafts, pulleys, belts and/or gears. Both motors have speed control systems and can be driven independently via a machine controller 907. The two drives are then linked via belts and pulleys and/or gears, with one of them linked to an overrun clutch. This clutch is set up such that when only motor 903 is driven in the direction of paper travel, both the feeding and receiving rollers will turn. However, if only motor 901 is driven in the direction of paper travel, only the feeding roller will turn. The method of operation is then: motor 901 is powered and turns the "feed rollers"; this drives a sheet of paper towards the "receiving rollers". As the paper approaches, motor 903 is powered and controlled to drive at the chosen speed. Motor 901 power may be reduced to ensure that, although it assists in the driving, it does not overtake motor 903, or power may be removed from motor 901. The "feed rollers" turn until the sheet has passed, and the motors are then stopped. With the arrival of the next sheet, motor 901 starts again, turning just the "feed rollers" to advance a second sheet of paper. This sheet is transported towards the "receiving rollers" and the awaiting first sheet which remains stationary, thus enabling the two sheets to be overlapped.

[0041] Thus, semi-independent drive of two separate drive systems can be provided such that the feeding roller can be driven forward without affecting the receiving roller. The use of a mechanical link incorporating an overrun clutch means that if the receiving roller is driven, the feed roller will also be driven at a speed defined by the gear/pulley ratios. Optionally, motor 901 can still be driven at this time, in order to take some of the load, as long as the power is not sufficient for it to overtake motor 903.

[0042] The exact drive train from each motor to its respective roller is unimportant, and the motors may be linked to the roller drive shaft by a series of gears, belts and pulleys, if desired. The overrun clutch can, instead, be put on the motor 901 drive shaft, but it must be mounted in reverse so that it performs the same function.

[0043] The clutched link introduces very few additional components, all of a low cost. The use of this clutch ensures that the speed is very accurately matched through the gear ratios without resorting to complex control systems with precision feedback. There is also no use of expensive electromechanical clutches with the associated risk of wear and failure that these can entail.

[0044] Importantly, although the feeding speeds of the feeding rollers and receiving rollers may be matched exactly, it is often desirable to have the two roller sets linked so as to be driven having a predetermined specific ratio between their feeding speeds. It may, for example, be desirable to set the feeding speed of the feeding rollers

slightly higher than the feeding speed of the receiving rollers in order to ensure that a buckle is normally formed in the sheet, in order to account for tolerances in roller diameter/feeding speed due to wear, manufacturing defects, etc.

[0045] Moreover, although it is necessary that the feed rollers 104 should be drivable independently of the receiving roller and belts 351, it is not absolutely necessary to employ two separate drive motors if suitable selectable drive chains and/or clutches are coupled to a single motor.

Claims

1. A sheet handling apparatus comprising:

a first sheet feeding means (104) for advancing a sheet along a sheet path;
a second sheet feeding means (351) for feeding the sheet along a continuation of the sheet path; means (901, 903, 905) for driving the first sheet feeding means independently of the second sheet feeding means and for common drive of the first and second sheet feeding means; and a drive controller arranged to control said driving means to enable a sheet to be driven along the sheet path from the first sheet feeding means to the second sheet feeding means whilst the second sheet feeding means (351) remains undriven and to enable continued feeding of the sheet by both sheet feeding means.

2. The sheet handling apparatus according to Claim 1, wherein for said common drive, the feeding speed of the first feeding (104) means is set at a specific ratio with respect to the feeding speed of the second feeding means (351).

3. The sheet handling apparatus according to Claim 1 or 2, wherein for said common drive the feeding speed of the first feeding means is matched with the feeding speed of the second feeding means.

4. The sheet handling apparatus according to Claim 1, 2 or 3, wherein the second sheet feeding means is stationary when undriven.

5. The sheet handling apparatus according to any preceding claim, wherein the first sheet feeding means comprises a feed roller (104).

6. The sheet handling apparatus according to any preceding claim, wherein the second sheet feeding means comprises one or both of a feed roller and a traction belt (351).

7. The sheet handling device according to any preced-

ing claim, wherein the driving means comprises:

a first motor coupled to the first feeding means;
a second motor coupled to the second feeding means; and
a coupling arrangement for selectively coupling the second motor to the first feeding means.

8. The sheet handling apparatus according to Claim 7, wherein the coupling arrangement is a series of gears, having appropriate ratios to set the relative feeding speeds of the first and second feeding means.

9. The sheet handling apparatus according to Claim 8, wherein the coupling arrangement comprises:

a feeding gear connected to moving parts of the first feeding means;
a receiving gear connected to moving parts of the second driving means; and
a belt or chain passing around both gears and constraining the feeding gear to rotate in synchronism with the receiving gear.

10. The sheet handling device according to any one of claims 7 to 9 wherein the coupling arrangement comprises a selectively operable clutch.

11. The sheet handling device according to claim 10 wherein said clutch is a one-way clutch.

12. The sheet handling device according to any preceding claim, wherein the second sheet feeding means forms part of an accumulation apparatus for accumulating a plurality of sheets fed to the accumulation apparatus by the first sheet feeding means.

13. A method of feeding sheets in which a sheet is fed by first feeding means to a second feeding means whilst the second driving means remains undriven and the sheet is further advanced by both the first and second feeding means set at a specific ratio therebetween.

14. A method according to Claim 13, in which the first and second feeding means are driven by respective drive shafts which are coupled together by means of a selectively operable coupling arrangement.

15. A method according to Claim 13 or 14, in which the speeds of the first and second feeding means are matched.

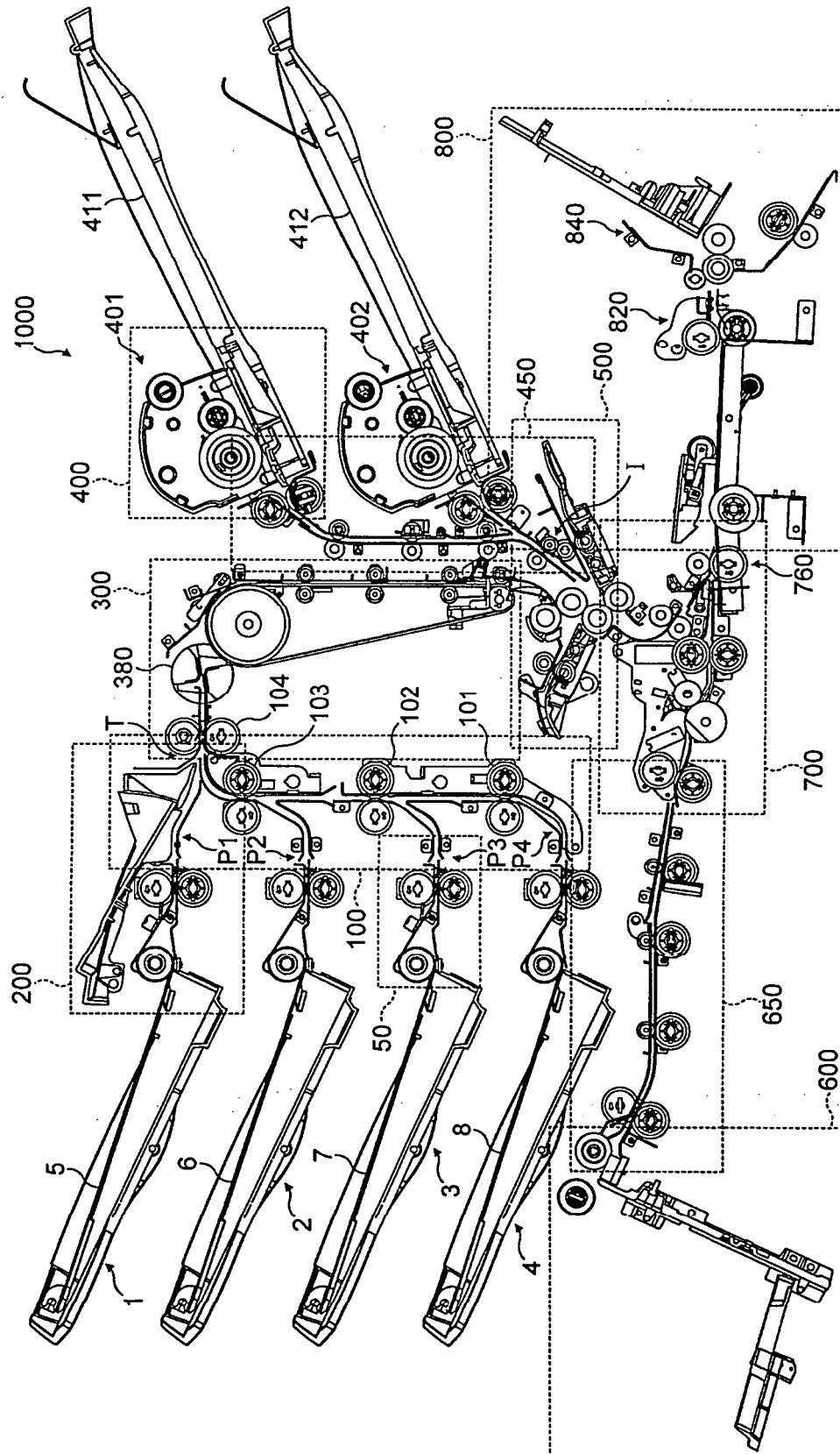


Fig. 1

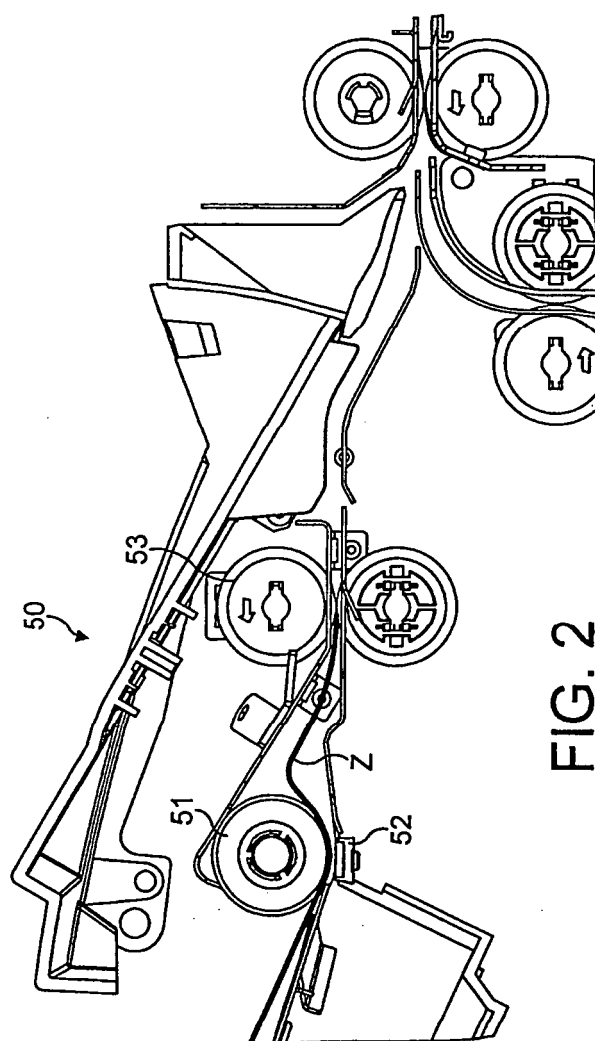
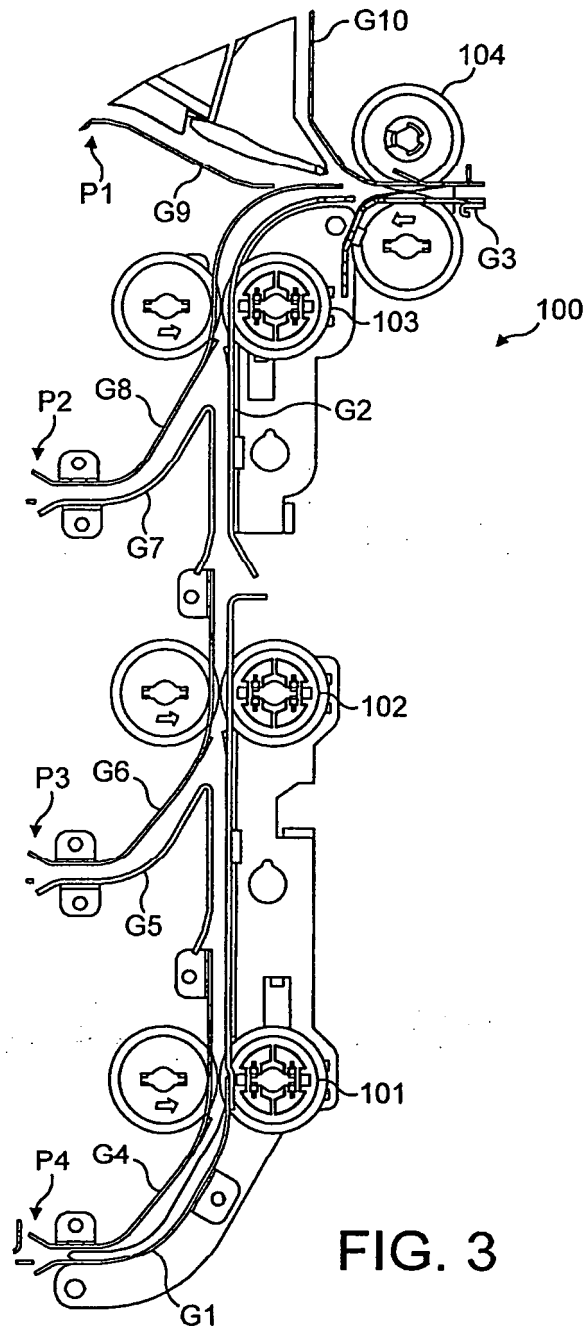


FIG. 2



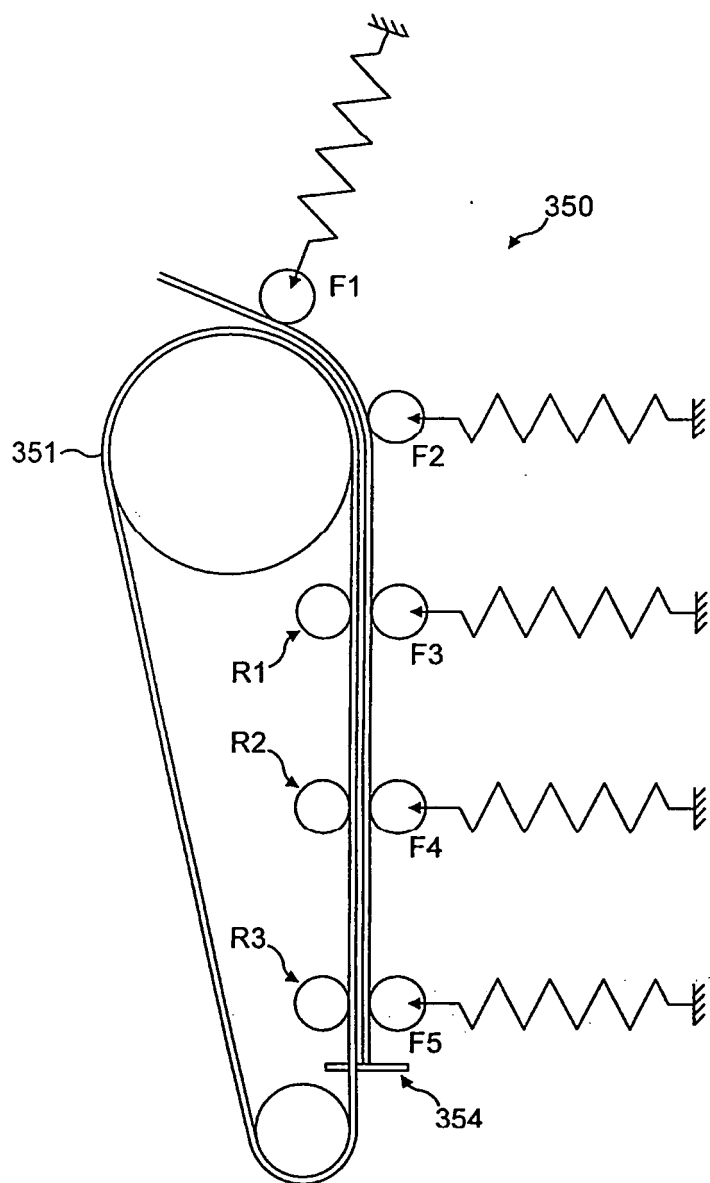


FIG. 4

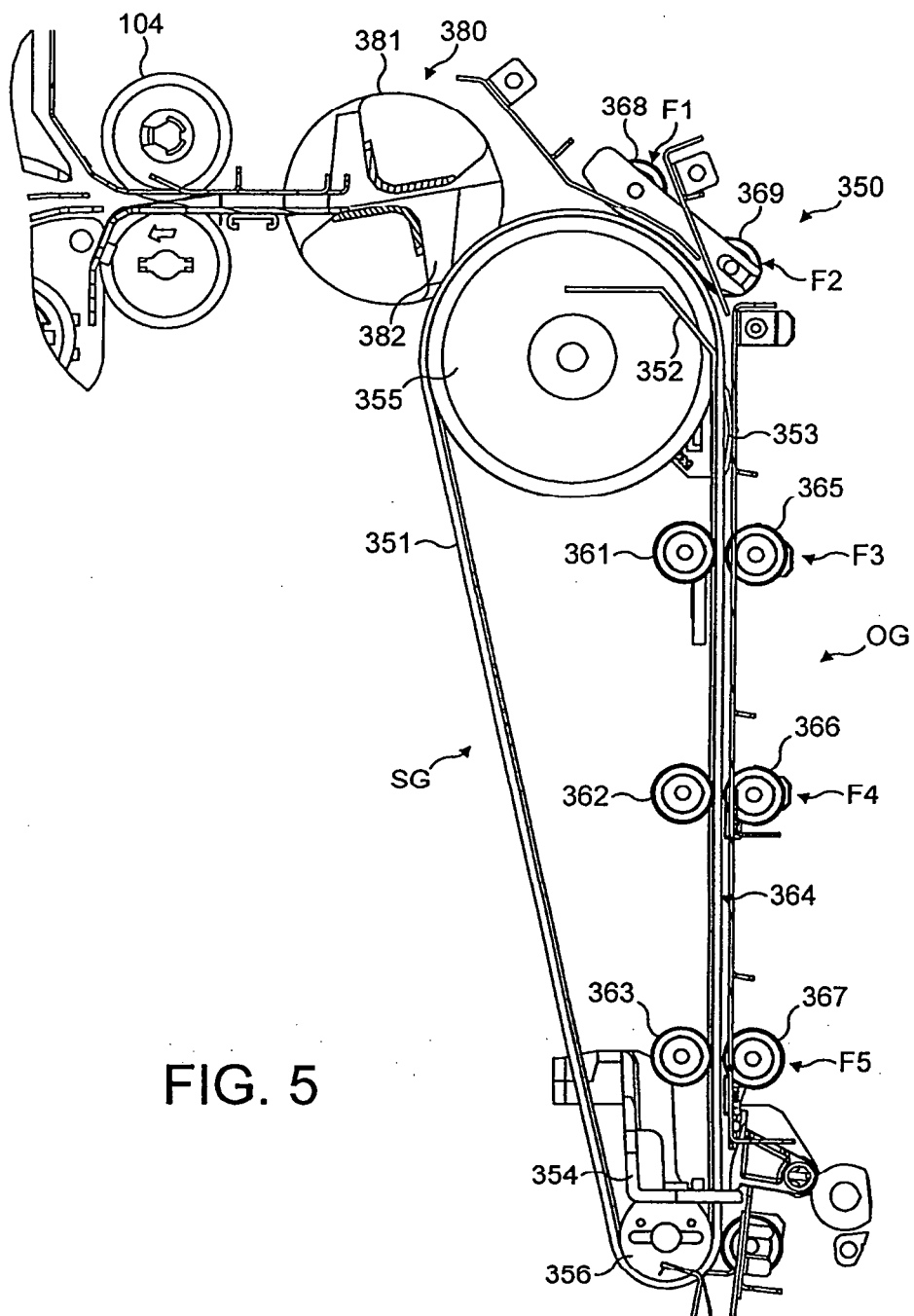


FIG. 5

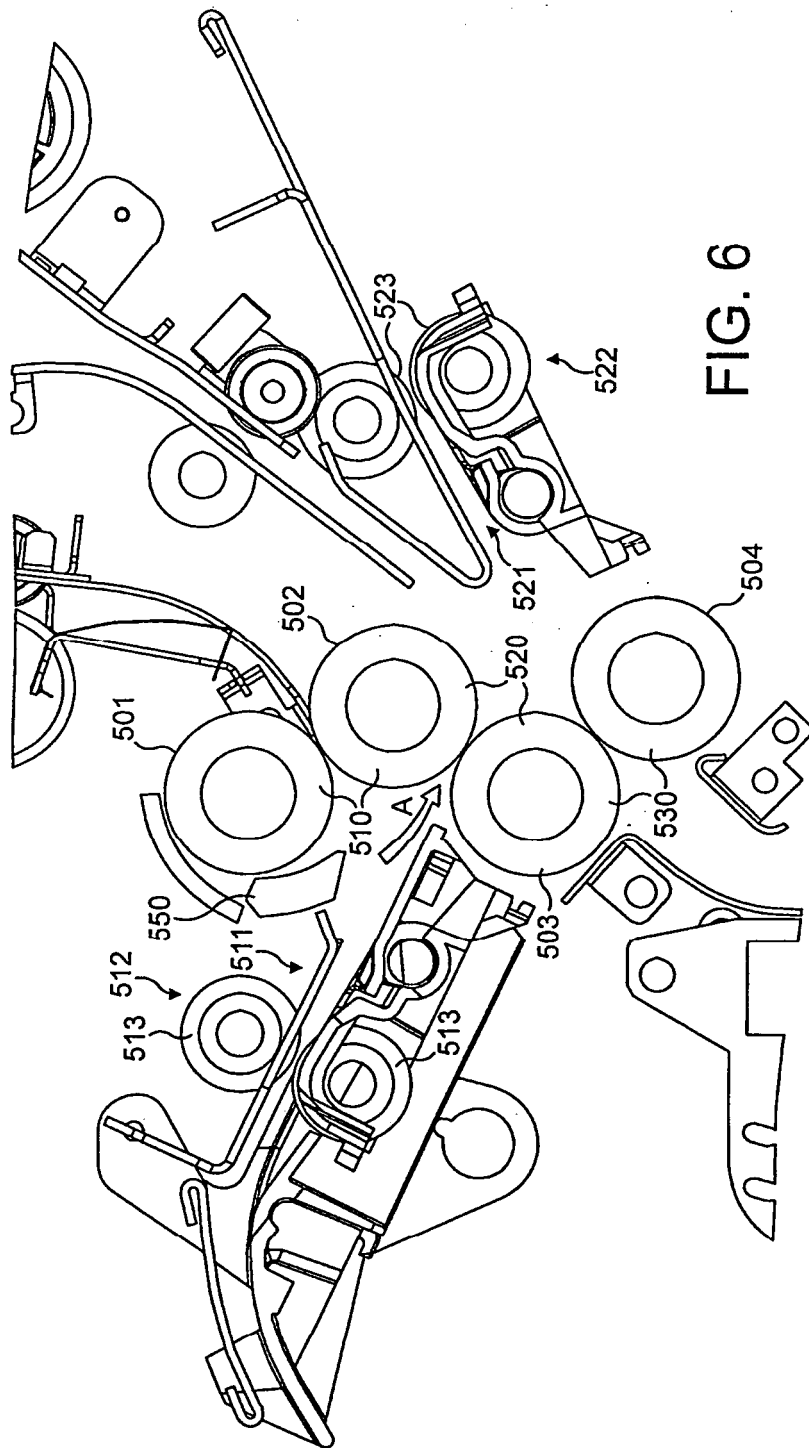


FIG. 6

