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Boot et al.

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[54] AUTOMATIC BOTTOM ROUGHING MACHINE

[72] Inventors: **Herbert W. Boot; Leslie R. Parr,**
both of Leicester, England

[73] Assignee: **USM Corporation, Flemington, N.J.**

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[58] Field of Search.....69/6.5; 12/17 R

[56]

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Primary Examiner—Alfred R. Guest

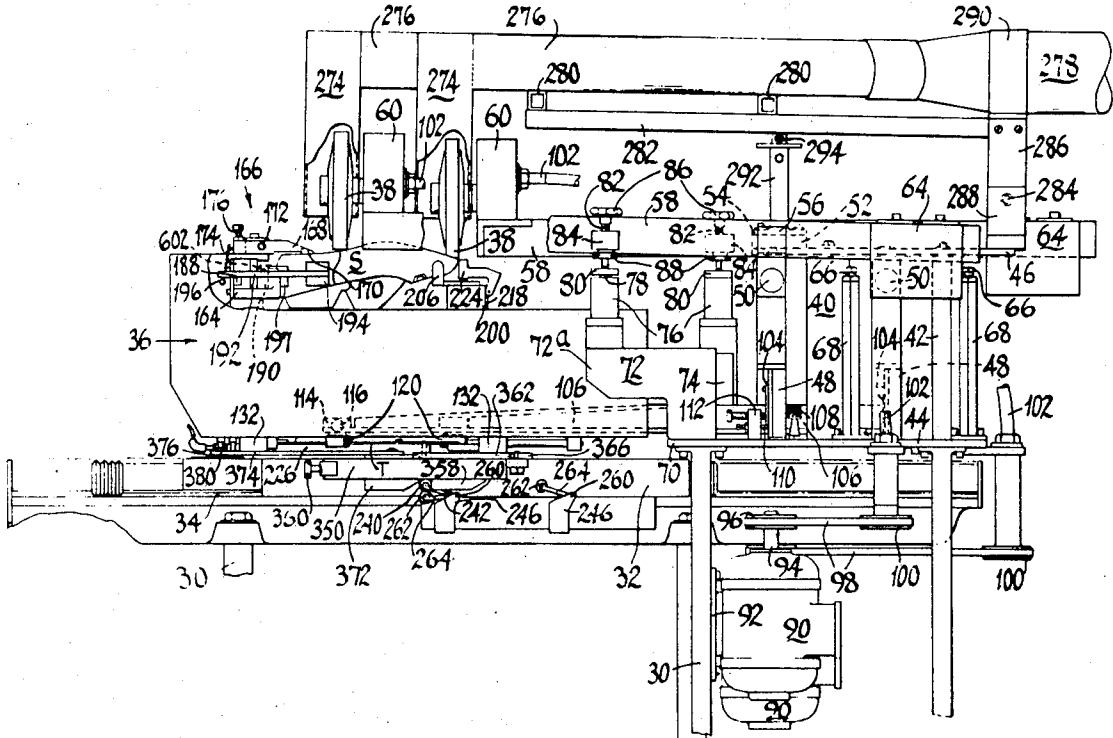
Attorney—Richard A. Wise et al.

[57]

ABSTRACT

An apparatus for operating on shoe bottoms having a pair of rotatable tools, a shoe support, means for moving the shoe support in advancing and returning directions relative to the tools, and means for arresting movement of the shoe support in the returning direction to a predetermined extent.

14 Claims, 10 Drawing Figures



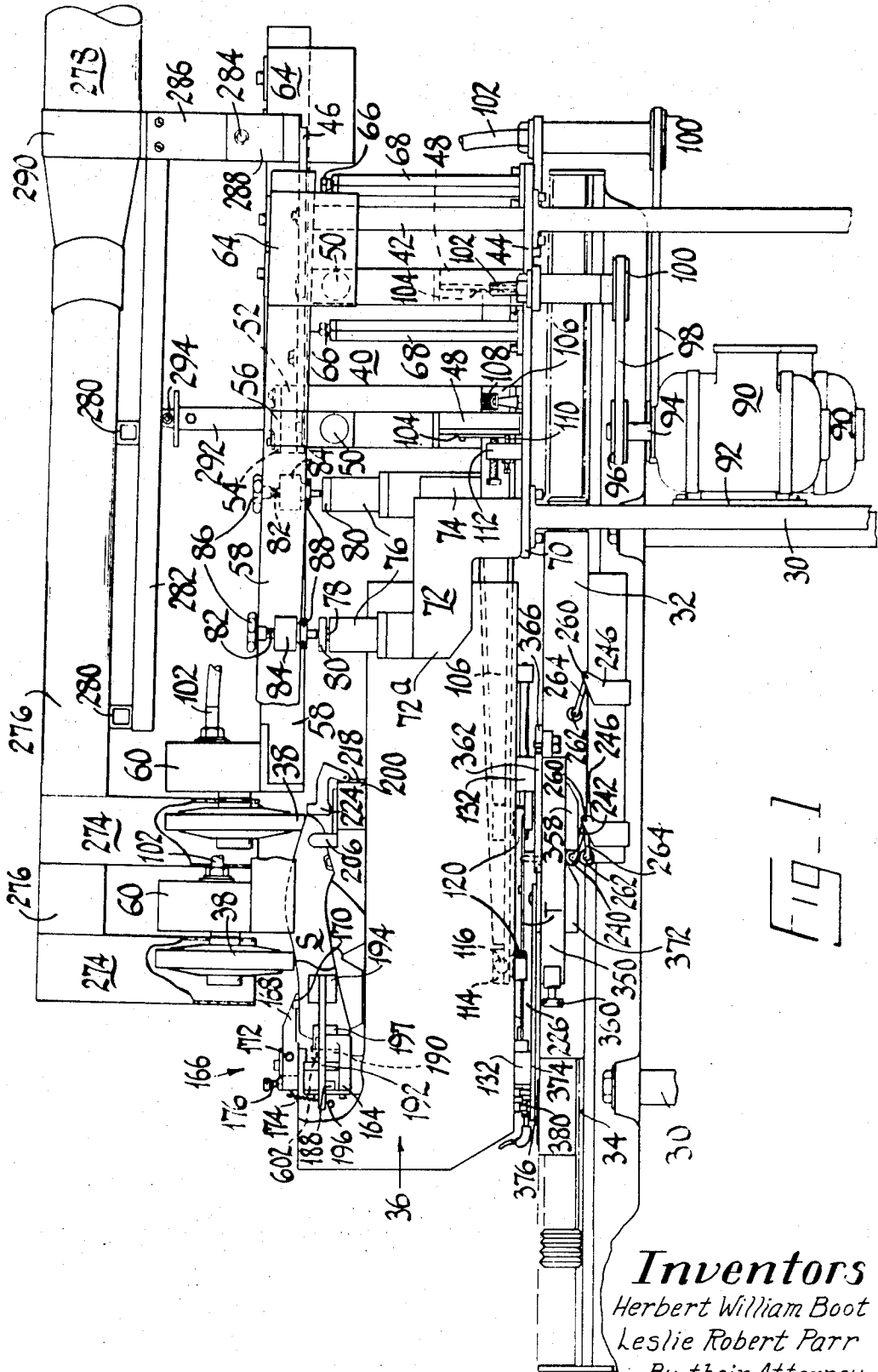
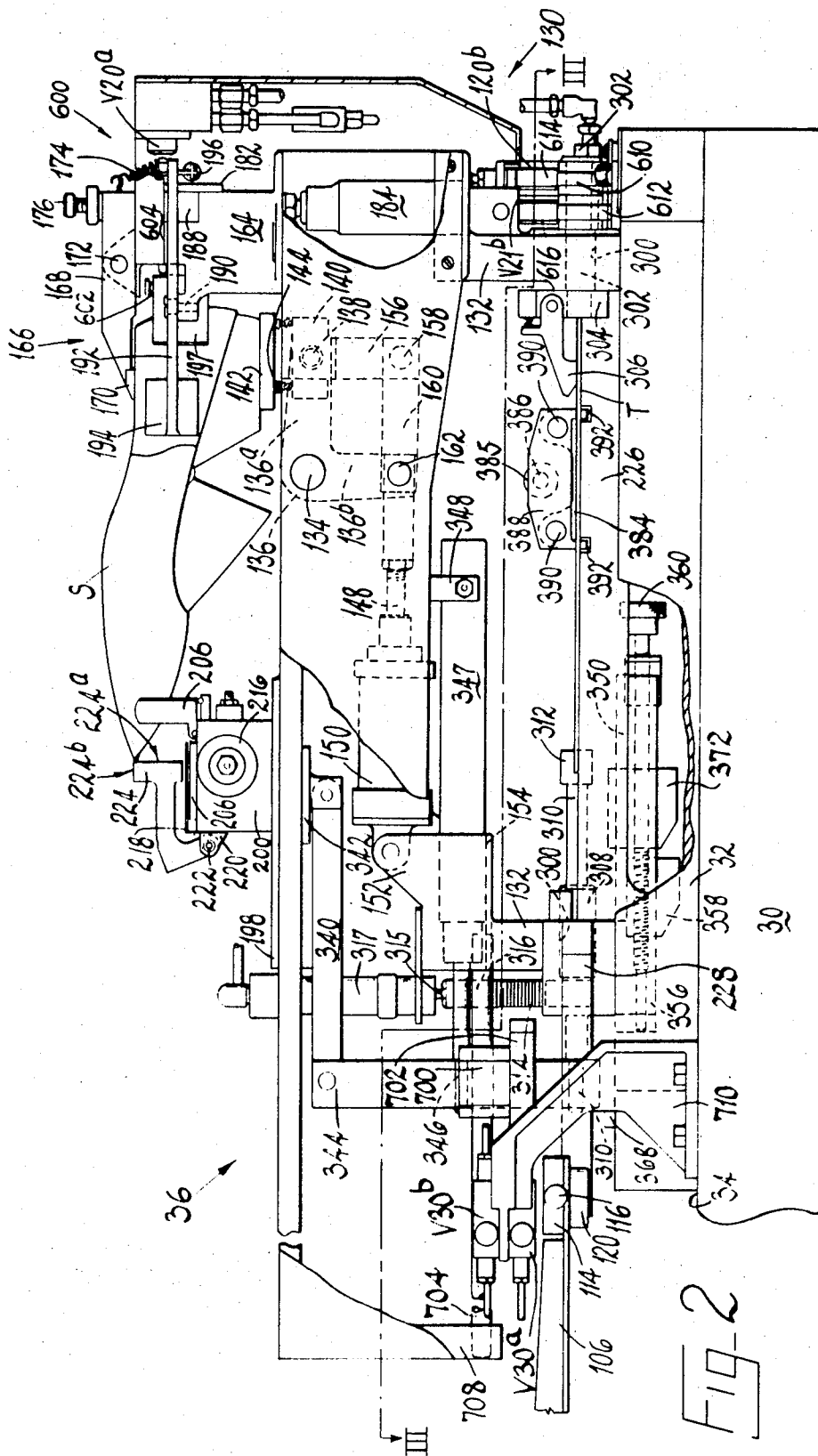
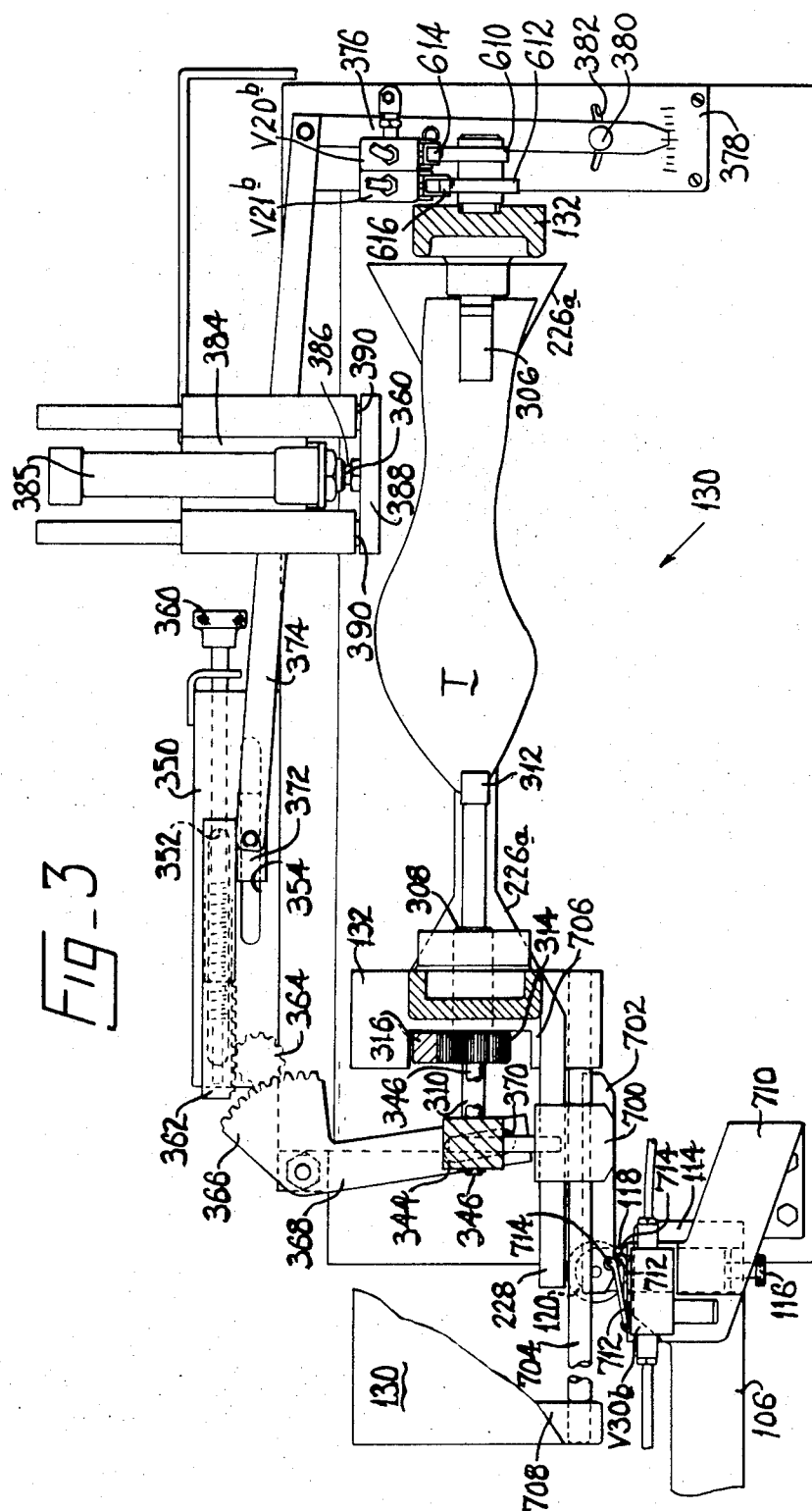
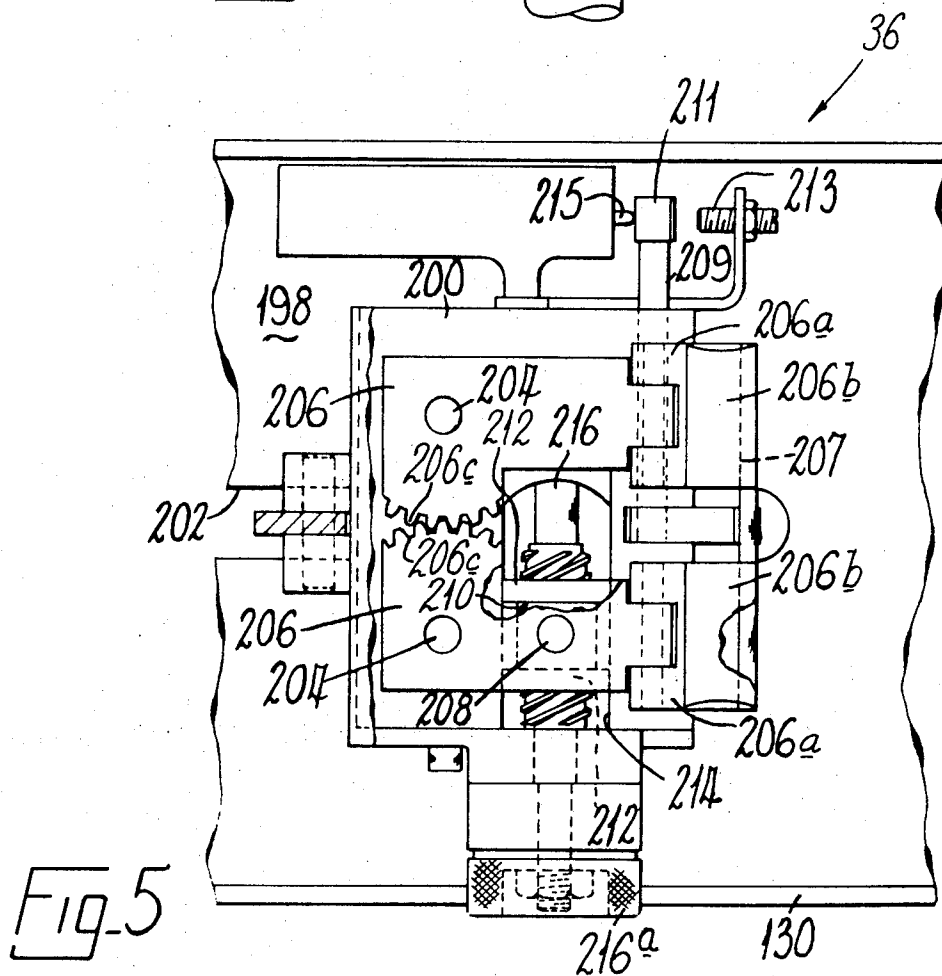
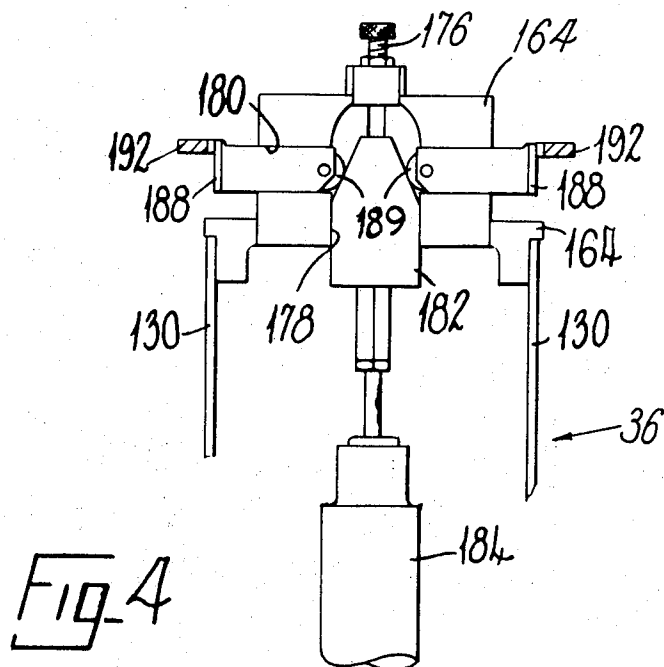


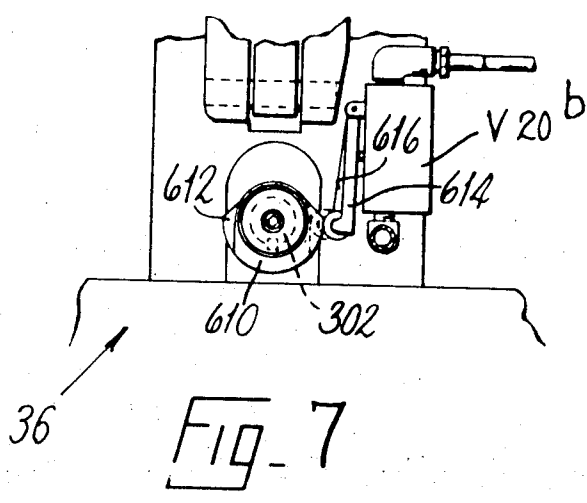
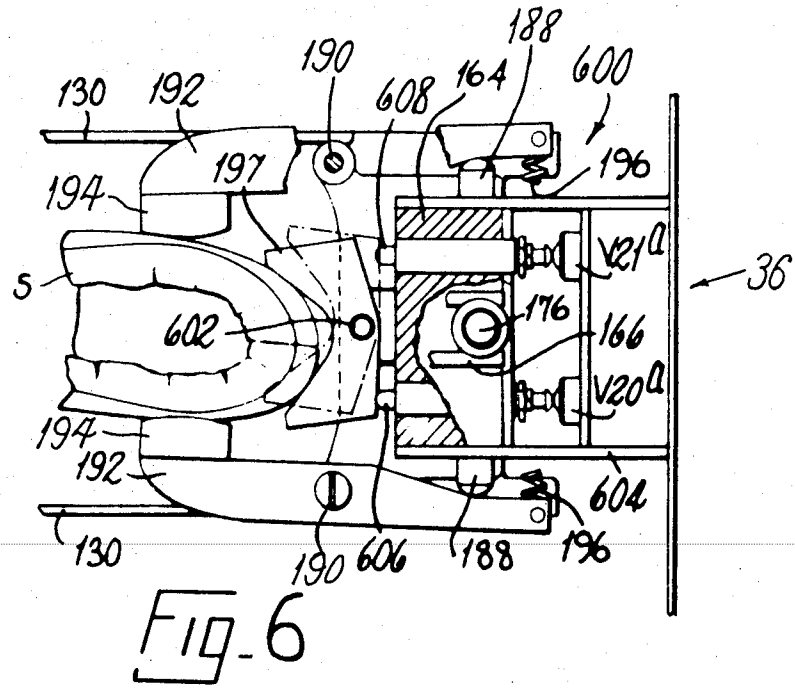
FIG. 1

Inventors
 Herbert William Boot
 Leslie Robert Parr
 By their Attorney
Cornelius H. Cleary









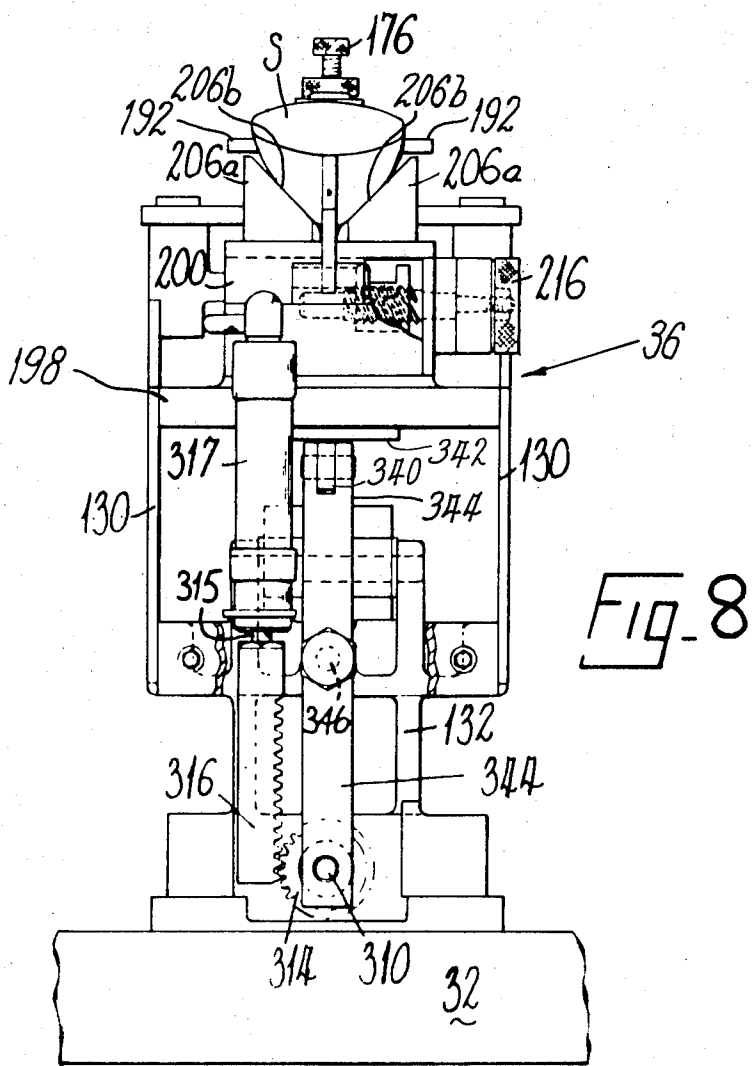
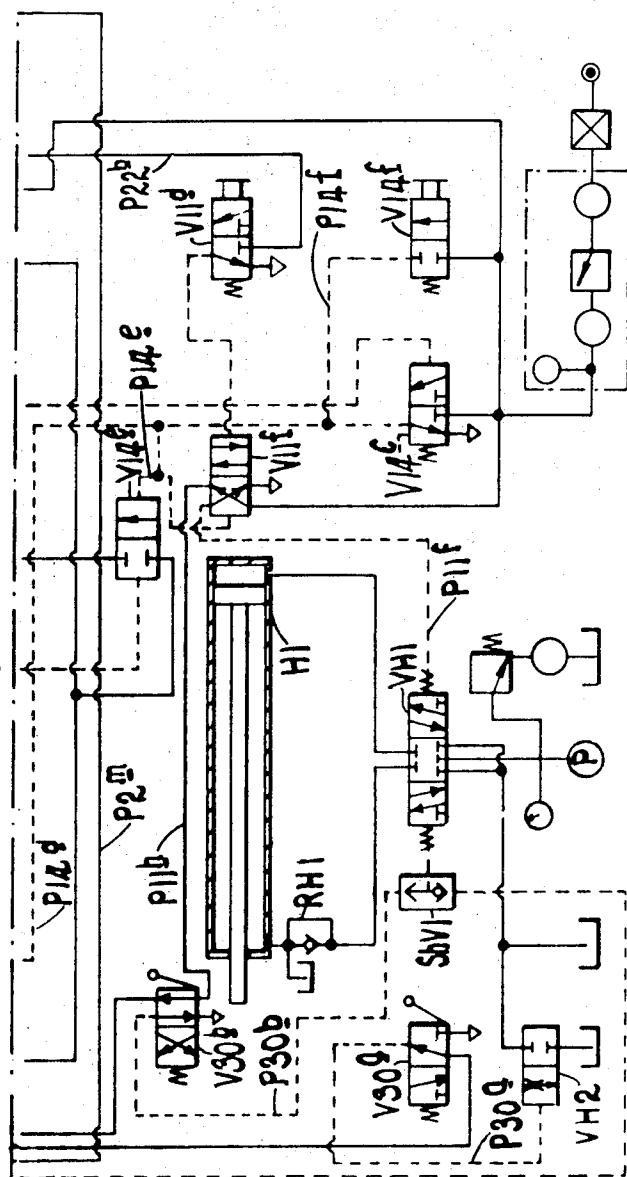


FIG-10



AUTOMATIC BOTTOM ROUGHING MACHINE

BACKGROUND OF THE INVENTION

This invention is concerned with improvements in or relating to apparatus suitable for use in the manufacture of shoes, more especially apparatus suitable for use in operating on marginal portions of shoe bottoms. The term "shoe" is herein used generically as indicating outer footwear generally and as including portions or components of outer footwear as, for example, assembled uppers and the like, in the course of manufacture.

Various apparatus suitable for use in performing a roughing operation on marginal portions of shoe bottoms have been proposed, for example, in U.S. Pat. No. 3,233,438 issued Feb. 8, 1966 to P. T. Hansen et al, and U.S. Pat. No. 3,518,851 issued July 7, 1970 to H. W. Boot et al, the latter pertaining to a modification of the apparatus disclosed in the earlier patent. Each of the apparatus disclosed above comprises a shoe support in the form of a box-like structure for supporting, bottom uppermost, a last carrying a shoe upper, tool supporting means comprising two support arms, mounted for pivotal movement about horizontal and vertical axes for supporting two rotary roughing tools in the form of radial wire brushes, fluid pressure operated means for moving the shoe support in a path extending generally lengthwise of a shoe supported thereby beneath the tools supported by the tool supporting means, the tools being supported in tandem relation along the path of the shoe support, and means, including two sensing devices, one associated with each tool, arranged to cooperate with cam means in the form of a template supported by template supporting means associated with the shoe support, for moving the tools widthwise of the path of the shoe support when the shoe support is moved along its path, in the operation of the apparatus, whereby the tools are caused to operate progressively along opposite marginal portions of the bottom of a shoe supported by the shoe support.

In the operation of the apparatus referred to above, after the shoe support has moved in one direction along its path to carry a shoe supported thereby beneath the tools as aforesaid, it is then returned in an opposite direction along said path to a loading/unloading position which is the same for all styles and sizes of shoe. The arrangement is such that this position is determined in relation to the end portion located remote from the tool supporting means, viz. the heel end portion of a shoe supported by the shoe support. Since, however, this position is the same for all styles and sizes of shoe to be operated upon, and this means that the largest size of shoe has to be accommodated, there will be excessive travel of the shoe support when operating on a small-sized shoe. In the course of a working day a substantial time is thus taken up by such ineffective travel of the shoe support when operating on shoes less than the largest size.

It accordingly is one of the various objects of the present invention to provide an improved apparatus suitable for use in performing operations on marginal portions of shoe bottoms wherein operating time is utilized more efficiently than in previous apparatus.

BRIEF DESCRIPTION OF THE INVENTION

The invention provides apparatus suitable for use in operating on marginal portions of shoe bottoms comprising a shoe support, tool supporting means, means for effecting relative movement between the shoe support and the tool supporting means in one direction whereby, when the apparatus is in use, a tool supported by the tool supporting means is caused to operate progressively along a marginal portion of a shoe supported by the shoe support, and thereafter in an opposite direction, and includes novel means whereby such relative movement in an opposite direction can be appropriately arrested. The arrangement is such that, when the apparatus is in use, the effective position of the arresting means is set according to the position in the shoe support of the end portion, nearer to the tool supporting means, of a shoe supported by the shoe support.

More specifically, the present invention comprises an apparatus for use in advantageously performing a roughing operation along opposite marginal portions of bottoms of shoe uppers. The apparatus includes a shoe support for supporting, bottom uppermost, a last carrying a shoe upper, tool supporting means for supporting two rotary roughing tools and fluid pressure operated means for effecting movement of the shoe support, along a feed path extending generally lengthwise of a shoe supported by the shoe support. When the apparatus is in use, tools supported by the tool supporting means are caused to operate progressively along opposite marginal portions of the bottom of a shoe supported by the shoe support. Thereafter, the shoe support is moved in an opposite direction by the operating means. The apparatus also includes means for supporting a template selected according to the style of shoe to be operated upon, and sensing means for cooperating with a template supported by the template supporting means. The template supporting means is arranged beneath a shoe supported by the shoe support and is mounted for movement with the shoe support. The sensing means is operatively connected with the tool supporting means and comprises two rolls arranged to engage with opposite edges of a template supported as aforesaid, each roll being connected to a tool supported by the tool supporting means. The sensing means is thus effective, in cooperation with a template supported by the tool supporting means to control the tool supporting means so that the tools follow a path corresponding to the shape of the marginal portions of the bottom of a shoe supported by the shoe support, as the latter carries the shoe beneath the tools as aforesaid.

The apparatus for this invention further includes arresting means whereby movement of the shoe support in said opposite direction can be properly arrested. The arrangement is such that, when the apparatus is in use, the arresting means is set according to the position in the shoe support of the end portion, nearer to the tool supporting means, viz. the toe end portion, of a shoe supported by the shoe support. The arresting means of the apparatus may comprise a switch arrangement including a control valve for controlling the flow of pressure fluid to the fluid pressure operated means for moving the shoe support as aforesaid, and an actuator arrangement for said switch arrangement, the actuator

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arrangement comprising an actuator member adjustably positioned on the shoe support in a position determined by the style and size of a selected template supported by the template supporting means. The switch and actuator arrangements of the arresting means are so positioned relative to one another that the control valve is arranged to be actuated by the actuator member, as the shoe support is moved in said opposite direction, when the toe end portion of a shoe supported by the shoe support has reached a "datum" line; with the toe end portion of the shoe at said "datum" line the shoe can be removed and a further shoe of similar size can be placed in the shoe support.

The apparatus may also include as an added feature decelerating means whereby movement of the shoe support in said opposite direction can be decelerated prior to being arrested as aforesaid. The decelerating means also may comprise a switch arrangement and an actuator arrangement therefor, the actuator arrangement comprising an actuator member secured to the switch actuator member for adjustment therewith, and the switch arrangement comprising a restrictor valve by which the flow of pressure fluid to said fluid pressure operated means for moving the shoe support is restricted.

The shoe support of the apparatus may include two devices by which opposite end portions, viz. the toe and heel end portions, of a shoe can be supported, the arrangement being such that the one of said devices by which the end portion, nearer the tool supporting means, as shown, the toe end portion, of a shoe can be supported, is mounted for adjusting movement relative to the other of said devices, viz. the heel end portion supporting device. The toe end portion supporting device of the shoe support is positionable or adjustable in the shoe support according to the size and style of a template supported by the template supporting means. When the apparatus is in use, the position of the toe end portion supporting device is adjusted at the same time as the switch arrangements of the arresting and decelerating means are positioned; a common locating arrangement may be provided for this purpose.

The construction and arrangement of the apparatus of the invention is such that the movement of the shoe support may be restricted to a minimum path length determined according to the size of shoe being operated upon. In this way excessive travel of the shoe support along the feed path may be minimized, thus insuring longer effective operating time in the course of a working day.

BRIEF DESCRIPTION OF THE DRAWINGS

The invention is hereinafter described with reference to the following drawings illustrating an embodiment of the apparatus of the invention in which:

FIG. 1 is a schematic front view in elevation, with portions broken away of the apparatus;

FIG. 2 is a schematic, rear view in elevation of parts of the illustrative apparatus showing the shoe support, template supporting means and locating arrangement of said apparatus;

FIG. 3 is a section taken along the line III—III of FIG. 2;

FIG. 4 is a fragmentary schematic end view of a heel clamp arrangement of the shoe support of the apparatus;

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FIG. 5 is a fragmentary schematic top view of a toe end portion supporting device of the shoe support of the apparatus;

FIG. 6 is a fragmentary schematic plan view of a heel end portion supporting device of the shoe support and the shoe sensing arrangement thereof of said apparatus;

FIG. 7 is a fragmentary schematic end view of the template sensing arrangement of the illustrative apparatus;

FIG. 8 is a schematic end view of the shoe support and the template supporting means showing details of the template rotating means of the illustrative apparatus; and

FIGS. 9 and 10 together form a diagram of a fluid pressure control circuit of the illustrative apparatus.

GENERAL ARRANGEMENT

The illustrative apparatus, which is an apparatus suitable for use in performing a roughing operation on marginal portions of shoe bottoms in preparation for the attachment of outsoles thereto by means of cement, comprises a frame 30 (of which upstanding frame members are shown), see FIG. 1, a table 32 mounted for sliding movement in a rectilinear path along an upper horizontal surface 34 of the frame 30 and carrying a shoe support generally designated 36 for supporting, bottom uppermost, a last carrying a shoe S to be operated upon, and means for mounting roughing tools in the form of two radial wire brushes 38 in tandem relationship along the path of movement of a shoe supported by the shoe support, so that in the operation of the illustrative apparatus the brushes 38 are caused to operate progressively along opposite marginal portions of the bottom of the shoe as the support 36 is moved as aforesaid.

BRUSH SUPPORTING MEANS

The brush supporting means of the illustrative apparatus comprises a column 40 (see FIG. 1) and an upstanding support member 42 both mounted centrally of the apparatus on a plate 44 secured on the frame 30 and in turn supporting, at the upper ends thereof, a plate 46. Mounted in bearings of pivotal movement in the plates 44, 46, one at either side of the frame 30, are two upstanding support members 48, a left-hand one of said members (viewing FIG. 1) being disposed forwardly of a right-hand one thereof and each member carrying a bracket 50 provided, at the upper end portion thereof, with a transverse pivot pin 52 on which is pivotally mounted a block 54. Each block 54 has fixedly secured thereon, by means of an angle member 56, an arm 58 carrying, at its left-hand end (viewing FIG. 1), a reduction gear box 60 an output shaft of which is operatively connected with one of the radial wire brushes 38, the mounting for the brushes 38 being such that they are readily removable from said shafts. At the opposite end of each arm 58 is secured a suitable counterweight 64, each arm 58 being so pivoted on its pin 52 that when a brush 38 is supported thereby as aforesaid, the weight of the brush is sufficient to cause the arm to pivot in a counterclockwise direction (viewing FIG. 1). When no brush is supported as aforesaid, the counterweight 64 causes the arm 58 to pivot in an opposite direction to an extent determined by an adjustable screw 66 carried at the upper end portion of an upstanding post 68 supported on the plate 44, there

being one post 68 mounted at each side of the frame 30.

The left-hand end (viewing FIG. 1) of the plate 44 is formed with two integral projections 70, one at either side of the plate, on which are mounted supporting brackets 72, 74, the left-hand bracket 72 having a forwardly projecting portion 72a. Mounted on the forwardly projecting portion 72a and on the right-hand bracket 74 are cylinders 76 in each of which is accommodated, for sliding heightwise movement under air under pressure, a piston (not shown), the pistons and cylinders constituting left-hand and right-hand piston and cylinder arrangements C13 and C14 (FIG. 9). Each piston has a piston rod 78 which is provided with an enlarged head portion 80 arranged to be engaged by a lower end portion, of reduced diameter, of an adjustable screw 82. The screws 82 are mounted in threaded bores formed in blocks 84 brazed onto the arms 58, each screw being provided with a knob 86 to facilitate turning and with a knurled locknut 88 threaded thereon to cooperate with the block 84 to secure the screw 82 in adjusted position.

The brushes 38 of the illustrative apparatus are caused to rotate by means of two electric motors 90, the arrangement of only one of which is shown in FIG. 1. Each is mounted on supports 92 depending from the frame 30, a drive shaft 94 of each motor carrying a pulley 96 connected by means of a belt 98 with a pulley 100 supported on the frame 30 and connected through a flexible drive cable 102 to the gear box 60. The brushes are caused to rotate in such a manner that as they each operate along a marginal portion of a shoe bottom as aforesaid, when the illustrative apparatus is in use, they effect an inward wiping action on said marginal portion.

Each support member 48 of the illustrative apparatus has secured thereto, at its lower end portion, a plate 104 to which is secured an end portion of an elongated arm 106. The arms 106 are urged inwardly towards one another, i.e. about the axes of the members 48, by means of springs 108, inward movement of each arm being limited by engagement of the plate 104 with an adjustable stop screw 110 (one only shown in FIG. 1) mounted in a block 112 secured on the table 44. Mounted at the end portion, remote from the plate 104, of each arm 106 is a block 114 (see also FIGS. 2 and 3) which in turn supports a slide 118 (see FIG. 3) for sliding movement transversely of the block 114. An adjusting screw 116 is provided for effecting such movement of the slide. The slide 118 carries at an underside thereof a roll 120 which is arranged, when the illustrative apparatus is in use, to engage with an edge portion of a template T, as will hereinafter be described.

The illustrative apparatus is also provided with dust extracting means comprising two hoods 274 (FIG. 1) which are arranged over the brushes 38, the hoods being mounted at left-hand end portions of tubes 276 which are connected, at their rearward end portions, to a tube 278 in turn connected, by means of a flexible hose (not shown) to a dust extraction unit (not shown). The tubes 276 are supported on cross arms 280 secured on arms 282 mounted, for pivotal movement about a cross shaft 284, on a support bracket 286, the shaft 284 being supported in a bracket 288 mounted on the plate

46. The tube 278 is also supported on the bracket 286 by means of a clip 290. Also supported on the plate 46 is an upstanding post 298 at the upper end of which is carried a stop pin 294 for limiting pivotal movement of the arms 282 in an anticlockwise direction (viewing FIG. 1) and thus determining the lowermost position of the hoods 274.

SHOE SUPPORT

The shoe support 36 (FIGS. 1 and 2 and also 4-8) of the illustrative apparatus comprises a box-like structure 130 supported on two support members 132 secured to the table 32 (FIGS. 1 and 2). Supported between side walls of the structure 130 is a cross shaft 134 (FIG. 2) on which are supported, spaced apart, two bell crank levers 136 (FIG. 2) each having a substantially horizontal arm 136a and a depending arm 136b. Pivotally mounted on a shaft 138 supported between end portions of the arms 136a of the bell crank levers is a block 140 on which is supported a plate 142, forming part of a heel end portion supporting device of the shoe support 36. Four springs 144 are provided which act between the block 140 and the plate 142 to urge the latter upwardly. Supported by the arms 136b of the bell crank levers is a short shaft (not shown) to which is operatively connected one end of a piston rod 148 (FIG. 2) of a piston (not shown) accommodated in a cylinder 150 which is pivotally mounted on a block 152 fixedly secured to a plate 154 part of the structure 130, the piston and cylinder 150 constituting piston and cylinder arrangement C11 (FIG. 9). Supported on the shaft 138 are upper end portions of two links 156, spaced at either side of the block 140, the lower end portions of the links supporting a shaft 158 to which end portions of two links 160 are connected. Opposite end portions of the links 160 are supported on shafts 162 mounted in the side walls of the structure 130. The bell crank levers 136 and links 156, 160 are so arranged that, upon actuation of the piston and cylinder arrangement C11, the block 140 and plate 142 supported thereon are caused to move heightwise of the structure 130.

Mounted above the plate 142 on a block 164 (FIGS. 1, 2 and 4) secured on the structure 130 is a foot generally designated 166 and comprising two arms 168 supporting a transverse shoe engaging member 170. The arms 168 are mounted for pivotal movement about a pin 172 supported in the block 164 and are urged in a counterclockwise direction (viewing FIG. 1) by means of a spring 174 acting between the block and the arms, an adjustable stop screw 176 being mounted in the arms for limiting counterclockwise movement thereof. The block 164 is also provided with a vertical bore 178 (FIG. 4) opening into a transverse bore 180 formed therein. A plunger 182 constituted by a piston rod of a piston (not shown) accommodated in a cylinder 184, which is fixedly secured in the structure 130 (the piston and cylinder constituting piston and cylinder arrangement C12 (FIG. 9)), is slidable in the vertical bore 178 and two slide blocks 188, each carrying a roll 189 at its inward end portion, are mounted in the traverse bore 180, one at either side of the vertical bore 178. Mounted on the block 164, one at either side of the foot 166, for pivotal movement about pins 190, are two arms 192 (FIGS. 1, 2 and 6) each of which carries a

heel clamping member 194 in the form of a plate. The ends of the arms 192 remote from the members 194 are connected by means of a spring 196 (FIGS. 1 and 6) and thus the members 194 are urged away from one another, forward end portions of the arms 192 being arranged to engage with ends of the slide blocks 188 whereby to limit movement of the members 194 away from one another (FIG. 6). Upon actuation of piston and cylinder arrangement C12, the plunger 182 is effective, through the rolls 189, to cause sliding movement of the slide blocks 188 in the transverse bore 180 of the block 164 whereby to cause the arms 192 to pivot about the pins 190. In this way the heel end portion of the shoe S positioned in the shoe support can be clamped by the members 194.

The box-like structure 130 is also provided with a cover plate 198 (FIGS. 2 and 5) on which a toe end portion supporting device, including a toe support block 200, is mounted for sliding movement fore and aft, a guide slot 202 (FIG. 5) being provided in the plate for the block. Supported on the upper surface of the block 200 are two shoe engaging members 206 mounted for pivotal movement about pins 204 mounted in the block 200 (FIG. 5). Forward end portions of the members 206 carry support elements 206a (FIG. 8) mounted for pivotal movement, about an axis extending transversely of the shoe support, and having inclined inwardly facing shoe engaging surfaces 206b (FIGS. 5 and 8) which form a V for accommodating a toe end portion of the shoe placed, bottom uppermost, in the shoe support. Mounted on the block 200 beneath the elements 206a and spring urged into engagement with the under-surfaces thereof is a bar 207 (FIG. 5) mounted for pivotal movement on a shaft 209 which extends transversely of the block 200 and projects therebeyond at one end. A depending arm 211 (FIG. 5) is secured at the projecting end and is arranged to engage with a stop screw 213 secured on the block by means of a bracket, whereby to limit upward movement of the bar 207, and to be swung, upon depression of the elements 206a, into engagement with a plunger 215 of a valve V11a (FIG. 9) also mounted on the block.

One of said members 206 has mounted therein a depending pin 208 which projects into a groove 210 formed in an upper surface of a slide block 212 mounted in a channel 214 formed therefor in the block 200 (FIG. 5). An adjusting screw 216, provided with a knurled head 216a, is captive in the block and threadedly engaged in the slide block 212 so that rotation of the screw causes movement of the slide block in the channel 214 and thus also pivotal movement of the right hand member 206. The members 206 are provided with rack portions 206c (FIG. 5) which extend inwardly towards one another and intermesh so that pivotal movement of the right hand member is effective to cause pivotal movement of the left hand member to take place also. Thus the surfaces 206b of the two members 206 can be moved towards and away from one another by rotation of the screw 216.

Also secured on the block 200 is an angle support member 218 (FIGS. 1 and 2) formed with a lug portion 220 on which is pivotally supported, on a pin 222, a toe end locating member 224 which extends forwardly over the block 200 and is provided with a flat toe end

engaging face 224a (FIG. 2). The member 224 is so constructed and arranged in relation to the brushes 38 of the illustrative apparatus that an upper face 224b (FIG. 2) thereof is located slightly below the lowest position of the brushes 38 and provides a gauge for determining the heightwise position of the shoe to be operated upon.

The block 164 also supports a V-shaped shoe heel engaging member 197 (FIGS. 1, 2 and 6) forming part of a shoe heel locating device generally designated 600, said heel clamp 197 being mounted for pivotal movement on a pin 602 (FIG. 6) between two positions, a first of which is shown in full line and a second in chain-and-dot line in FIG. 6. Supported in the block 164 adjacent the clamp 197 is a shoe sensing arrangement constituted by two shoe sensing devices comprising two plungers 606, 608 (FIG. 6) a first one, 606, of which is depressed when the heel clamp is moved to its first position and a second one, 608, of which is depressed when the heel clamp is moved to its second position. Both plungers 606, 608 are urged towards the shoe (to the left, viewing FIG. 6) by springs (not shown) mounted on the plungers and acting between them and the block 164. The plungers are arranged, when depressed, to actuate pneumatic valves V20a, V21a, respectively. Thus when a left shoe is placed in the illustrative shoe support (as shown in FIG. 6), the heel clamp is moved into its first position and thus, through the plunger 606, valve V20a is actuated. Similarly when a right shoe is so supported, valve V21a is actuated.

TEMPLATE SUPPORTING MEANS

The illustrative apparatus also comprises means for supporting the template T beneath the shoe S supported by the shoe support 36. Each support member 132 of the shoe support 36 is provided with a bore 300 (FIGS. 2 and 3), located on an axis extending lengthwise of the support. Accommodated within the bore 300 formed in the right-hand support member 132 (viewing FIG. 2) is a short shaft 302 which supports one end of a casting 226, extending between the two support members, and also a support block 304, formed integral with the casting, for a latch member 306 which is spring urged in a counterclockwise direction (viewing FIG. 2), into engagement with a template T when one is placed in operating position, a slot being provided in said support block 304 for receiving a projecting portion of the template to facilitate accurate location of the latter. Accommodated within the bore 300 formed in the left-hand support member 132 (viewing FIG. 2) is a sleeve 308 on which the other end of the casting 226 is supported and which supports a shaft 310 (FIG. 8) accommodated within a channelled portion in the casting, the shaft 310 being provided at one end with a slotted nosepiece 312 for engaging with a notch formed in the template. The shaft 310 also has fixedly secured thereon a pinion 314 (FIG. 8) which meshes with a vertical rack member 316 secured on a piston rod 315 which is movable heightwise of the shoe support and is accommodated with its piston (not shown) in a cylinder 317 (the piston and cylinder 317 constituting piston and cylinder arrangement C15). As will be hereinafter described in detail, heightwise movement

of the rack member 316 is effective to cause the casting 226, and also the template T supported thereby, to be rotated about the axis of the bores 300.

The casting 226 is provided at its ends with flared portions 226a (see FIG. 3) which facilitate control of the rolls 120, and thus of the brushes 38, when these approach the template and shoe, in the operation of the apparatus, and also when the rolls run off the template after having tracked along opposite sides thereof. In addition, a guide member 228 is provided, secured to the left hand flared portion 226a (viewing FIG. 3) and is arranged to be engaged by one of the rolls 120, as shown in said figure, when the shoe support is in a loading position.

Mounted on an extended end portion, remote from the template, of the short shaft 302 are two cam members 610, 612 (FIGS. 2, 3 and 7) forming part of two template sensing devices. When the template supporting means, and thus the template T, are in a first position, a first one 610 of said cam members causes a pivoted arm 614 to be depressed (as shown in FIG. 7) to actuate valve V20b and, when in a second position, a second one 612 of said members causes a pivoted arm 616 to be depressed to actuate valve V21b. In the illustrative apparatus the template sensing devices are so arranged that when the template supporting means is in its first position the template T supported thereby is arranged to enable a roughing operation to be performed on a left shoe and in its second position on a right shoe.

TEMPLATE AND SHOE SENSING ARRANGEMENTS (AND TEMPLATE REVERSAL)

In operating the illustrative apparatus the operator first ensures that a selected template T corresponding in respect of size and style to the shoe to be operated upon is located in the template supporting means. In the illustrative apparatus, templates are exchanged when the casting 226 is in the position shown in FIG. 2, that is, when the template supporting means is in its first position and thus the apparatus is set for an operation on a left shoe. The operator then places a shoe in the shoe support, the toe end portion thereof thus operating valve V11a (see FIGS. 9 and 10), and then operates manually operable valve V11b to allow air under pressure to operate valve V11c. Air from main pressure line P1m thus passes through valve V11c to piston and cylinder arrangements C11 and C12 whereby in sequence the plate 142 urges the shoe against the foot 166 and the arms 192 against the sides of the shoe.

Assuming that the shoe is a right, with the indicator 394 (FIG. 9) indicating that the template is set for a "right" operation and with valve V21b actuated by the cam 612, engaging of the heel end portion of the shoe with the heel clamp 197 (FIG. 6) causes the heel clamp 197 to be moved to its second position and thus causes valve V21a to be actuated. Air under pressure is thus admitted, from the line leading to piston and cylinder arrangement C12, along line P19, through valve V21a, along line P21, and through valve V21b to shuttle valve V22a. Air under pressure passing through valve V22a operates pilot valve V22b to allow air under pressure from valve V11b to pass along line P22b to valve V11d. The illustrative apparatus is now ready for a cycle of operation to be performed upon manual operation of valve V11d by the operator.

If the template T is set for a "left" operation, with valve V20b operated as aforesaid, when a left shoe is placed in the shoe support, valve V20a is also operated to allow air under pressure to pass from line P19 along line P20, through valve V20b, to shuttle valve V22a and thus to enable pilot valve V22b to be operated as aforesaid. If, however, the template is set for a "left" operation and a right shoe is placed in the shoe support or vice versa no air will be allowed to pass to valves V22a and V22b and thus valve V11d will not operate to initiate a cycle of operation of the illustrative apparatus.

Towards the end of a cycle of operation of the illustrative apparatus, valve V14b is tripped as the second brush 38 reaches the end of its traverse to cause the brush to be raised and also to cause the shoe support to return to its initial position; operation of valve V14b also allows air under pressure to pass along line P23a through shuttle valve V23a to cause pilot valve V23b to move from its position shown in FIG. 9 to allow air under pressure to pass therethrough along line P23b, through valve V23c and along line P23c to pilot valve V23d which is thus moved from its position shown in FIG. 9. When, as described below, later in the cycle valve V14b is re-set upon actuation of valve V30b, along line P23d through valve V23d and along line V23f to the upper end of piston and cylinder arrangement C15 and the indicator 394, thus to turn the template T through 180° and change the position of the indicator to indicate the opposite shoe. At the same time air from line P23f passes along pilot line P23g to valve V23c to cause it to move to the right (viewing FIG. 9) in readiness for the next cycle of operation of the illustrative apparatus.

In the next cycle of operation, the same sequence of operations takes place except that the valve V23c is now positioned to allow air under pressure to cause valve V23d to move back to its position shown so that air from line P23d now passes along line P23h to the lower end of piston and cylinder arrangement C15 and to the indicator 394 and also along pilot line P23i to move valve V23c back to its position shown.

Thus for successive signals received from valve V14b the template T is caused to be reversed through 180°. If, as above referred to, the template setting and a shoe placed in the shoe support do not correspond with one another, the template T is reversed by operation of manually operable valve V23e which allows air under pressure to pass along line P23j to valve V23a.

It will thus be apparent that the illustrative apparatus is prevented, when the template T is in one setting, from operating on an opposite shoe, means being provided, however, whereby the template can be simply and quickly reversed so as not to impede longer than necessary the operation of the apparatus.

TOE END PORTION SUPPORTING DEVICE LOCATING FROM THE TEMPLATE

The illustrative apparatus also includes means whereby the block 200 of the toe end portion support device can be accurately located lengthwise of the shoe support by means of the template T positioned therebeneath. For this purpose a link 340 (FIGS. 2 and 8) is connected at one end to a lug 342 formed on the undersurface of the block 200, and, at the other, to an upper end of an upstanding arm 344 the lower end of

which receives a left-hand end portion, of reduced cross section, of the shaft 310. Secured at an intermediate portion of the arm 344 is a piston rod 346 which is connected to a piston accommodated in a cylinder 347 (FIG. 2) (the piston and cylinder constituting piston and cylinder arrangement C16), the cylinder 347 being supported, at its left-hand end (viewing FIG. 2) on the plate 154 and at its right-hand end by brackets 348 secured to the side walls of the structure 130. Thus on operation of the piston and cylinder arrangement C16 the block 200 is moved along with the rod 310 with its nosepiece 312 for engaging template T. Movement of the block 200 towards the heel support is limited by engagement of the nosepiece 312 with the template whereby the block 200 is accurately located lengthwise of the shoe.

BRUSH PRESSURE VARIATION

Secured on the side of the table 32 is a block 350 (FIGS. 1 and 3) including a pair of slots 352, 354. Mounted for sliding movement in the slot 352 is a support member 356 (FIG. 2) for a cam member 358 (FIGS. 1 and 2), an adjusting screw 360 being captive in the support member 356 and in the block 350 and threaded into the cam member 358 for adjusting the position of the latter relative to the support member 356. The support member 356 is mounted on the under-surface of a rack member 362 (FIGS. 1 and 3) slidable on the upper surface of the block 350 and arranged to mesh with a pinion 364 (FIG. 3) supported on the block 350. The pinion 364 is arranged to mesh with a gear segment 366 formed integral with one arm of a bell crank lever 368 pivotally mounted on the table 32. The other arm of the bell crank lever 368 is provided with a slot 370 by which it is connected to the lower end portion of the arm 344. Thus movement of the arm 344, upon operation of the piston and cylinder arrangement C16, is further effective to cause the bell crank lever 368 to pivot and, through the gear segment 366, pinion 364 and rack 362, to move the support member 356 and cam member 358, thus to position the latter according to the size of shoe to be operated upon. Arranged in the path of the cam member 358 and mounted on pins 260 which are secured in brackets 246, are two rolls 262 (FIG. 1). The rolls are supported on trip arms 264 pivotal movement of which under the influence of the cam member 358 is effective to operate, in sequence, valves V13a and V14a (FIG. 9) which are effective, through piston and cylinder arrangements C13 and C14, to cause control of the pressure applied by the brushes 38 to be switched from valve RV13a to valve RV13b whereupon the brushes are caused to exert increased pressure on the shoe bottom, as will be hereinafter described. The positioning of the cam member 358 as aforesaid, and thus the timing of the variation in pressure applied to the brush supporting arms 102 may be so arranged that, in an operation on a ladies' high heel shoe, when the highest region of the shoe bottom is reached by the brushes 38, control of the pressure applied thereby is switched from valve RV13a to RV13b. Fine adjustment of such positioning for different styles of shoe is effected by the adjusting screw 360. Alternatively, the variation in pressure may occur, in an operation on a so-called "flattie" shoe, when the heel seat portion of the shoe bottom is reached by the brushes.

In the other slot 354 formed in the block 350 is slidably mounted a second cam member 372 (FIGS. 1, 2 and 3) which is pivotally mounted at one end of a link 374 (FIG. 3) the other end of which is connected to one end of a lever 376 (FIG. 3) pivotally mounted on the upper surface of an extension of the table 32. The other end of the lever 376 is in the form of a pointer which moves over a scale 378 (FIG. 3), a clamping screw 380 and slot 382 being provided for securing the lever, and thus the cam member 372, in adjusted position. In the path of the cam member 372 are arranged two rolls 240 (FIG. 1) which are mounted on trip arms 242 arranged to pivot about pins (not shown, but coincident with the pins 260) secured in the brackets 246. Pivotal movement of the arms 242 under the influence of the cam member 372 is effective to operate in sequence, valves V13b and V14b (FIG. 9) which, as hereinafter described, are effective, through piston and cylinder arrangements C13 and C14, to cause control of the pressure applied by the brushes 38 to be switched from valve RV13b to valve RV13c, whereupon the brushes are raised out of shoe engaging position. Thus, by the setting of the cam member 372, the timing of the lifting of the brushes 38 out of engagement with the shoe bottom is determined. In this way it is possible, for example, to select either an "all round" roughing operation or a "breast-line to breast-line" operation.

DECELERATING AND ARRESTING MEANS FOR DECELERATING AND ARRESTING RETURN MOVEMENT OF THE SHOE SUPPORT

The illustrative apparatus includes arresting means whereby the movement of the shoe support 36 to carry the shoe S from beneath the brushes 38 can be arrested, the shoe support 36 coming to rest at the end of a cycle of operation of the apparatus, in a loading/unloading position determined according to the style and size of shoe supported by the shoe support. In the operation of the illustrative apparatus, a "datum" line is established to which the end portion, nearer to the brushes 38, of a shoe supported by the shoe support (viz. the toe end portion) is located when the shoe support 36 is in said position, regardless of the style and size of such shoe. To achieve this datum line, the loading/unloading position of the shoe support for any given style and size of shoe is determined with reference to the template T corresponding to said shoe. Secured to the upstanding arm 344, which forms part of the locating arrangement for locating the block 200 of the toe end portion supporting device, are two actuator members in the form of cam members 700, 702 (FIGS. 2 and 3) mounted one above the other. The upper cam member 700, which forms part of the arresting means, is mounted for sliding adjusting movement, together with the arm 344, along a rod 704 (FIG. 3) secured at one end in a lug 706 formed on the left-hand support member 132 (viewing FIGS. 2 and 3) and at the other end to an end plate 708 forming part of the structure 130. The lower cam member 702, which forms part of the decelerating means, is secured to the member 700 for adjusting movement therewith. Supported by a bracket 710 mounted on the frame 30 are two valves V30a, V30b arranged one above the other. Valve V30a forms part of the decelerating means and valve V30b part of the arresting means. The valves are

each provided with a trip arm 712 which carries a cam member engaging roll 714, the rolls 714 of valves V30a and V30b lying respectively, in the paths of cam members 702 and 700.

In the operation of the illustrative apparatus, as the shoe support is caused to return to its loading/unloading position, after a roughing operation has been performed on the bottom of a shoe supported thereby, firstly the cam member 702 engages and actuates valve V30a to move it into its position shown in FIG. 10, from which position it is urged by a spring when not engaged by the cam member. Actuating valve V30a admits air under pressure from main line P1m along a line P30a to switch hydraulic valve VH2 from its shut-off position (shown in FIG. 10) to a restricted bleed-off position, so that fluid under pressure from hydraulic pump P is bled off and thus causes deceleration of the shoe support to be effected.

As the shoe support continues its return movement the upper cam member 700 engages and actuates valve V30b which is thus returned to its position shown in FIG. 10. Actuating valve V30b causes line P30b to be exhausted and hydraulic valve VH1 is thus returned to a central, shut-off, position so that movement of the shoe support is arrested.

It will be appreciated that since the position of the upstanding arm 344 varies according to the style and size of the template T supported by the template supporting means, the position of the cam members 700, 702 will also be thus varied. Since the datum line referred to above is determined by the actuation of valve V30b (actuation of which causes the return movement of the shoe support to be arrested), it is apparent that the datum line is determined according to the particular template T carried by the template supporting means, said template being, of course, selected according to the style and size of shoe being operated upon.

TEMPLATE EJECTION

When it is desired to exchange one template for another, piston and cylinder arrangement C16 is operated to cause the rod 310 to be withdrawn and template ejecting means is then brought into operation. This means includes a block 384 (FIGS. 2 and 3) mounted on the frame 30 and supporting a cylinder 385 forming part of piston and cylinder arrangement C17 the piston (not shown) of which is connected to a piston rod 386 carrying an ejector member 388. Two rods 390 are connected to the ejector member and slidable in bores formed in the block 384 for guiding the ejector member 388 and grooves 392 (FIG. 2) are formed in the casting 226 to allow the passage of depending portions of the member 388 during an ejection operation.

When the operator wishes to change templates, he first has to ensure that the template supported by the template supporting means is in a position on top of the casting 226 (as shown in FIG. 2) so that it can be readily removed. For the template T to be in this position valve V23d has to be shifted to the right (viewing FIG. 9) from its position shown in that figure. (It will be appreciated that valve V23d alternates between its two positions in successive cycles of the illustrative apparatus.)

Having established that the template is supported as desired, the operator operates treadle operated valve V16a to allow air along pilot line P16b to reverse valve V16b and thus allow air from main line P1m along lines P16c and P16d to reverse the flow of air to piston and cylinder arrangement C16 and to operate valve V17a. Operation of valve V17a allows air from accumulator A2, which is pressurized through valves V11f and V30b from main line P1m, along pilot line P17a to operate valve V17b whereby the flow of air to piston and cylinder arrangement C17 is reversed and the ejecting means thus operated. When the air from the accumulator A2 is exhausted, valve V17b is returned to its position shown in FIG. 9 and the ejecting means is withdrawn. Operation of valve V16a is also effective, through pilot line P16e and shuttle valve ShV1, to operate three-position hydraulic valve VH1 to move it to the right (viewing FIG. 10) from its central position. In this way, hydraulic fluid is admitted from pump P through said valve VH1 to the left hand end (viewing FIG. 10) of piston and cylinder arrangement H1 thus to cause the shoe support to be moved to the limit of travel thereof away from the tool supporting means. Thus the largest size of template can be accepted in the template supporting means.

OPERATION

A cycle of operation will now be described with particular reference to FIGS. 9 and 10.

Conveniently in a cycle of operation of the illustrative apparatus, the operator first ensures that a correct template T is mounted in the template supporting means and that such template is correctly oriented according to whether the apparatus is set for a next operation to be performed on a left or right shoe, as indicated by the indicator 394. If the template does not match the shoe next to be operated upon, it is ejected by the template ejecting means and replaced, as above described.

When the shoe support is in its loading/unloading position, valve V30b is held in the position shown in FIG. 10 and thus allows air under pressure from main line P1m via valve V11f to pass along line P30c, through valve V16a and along line P16a to hold valve V16b in its position shown in FIG. 9 whereby air under pressure is allowed to pass therethrough to the lower end of piston and cylinder and arrangement C16 and also into line P2m. The operator also ensures that regulator valves RV13a, RV13b and RV13c are correctly adjusted to the desired pressures.

The operator then takes the shoe and places it, bottom uppermost, with its toe in the V formed by the elements 206a of the toe support and with its heel on the plate 142. As stated above, placing the toe of the shoe causes valve V11a to operate to allow air from line P2m therethrough to manually operable valve V11b which the operator actuates when the shoe has been positioned in the shoe support. Operation of valve V11b allows air to pass through pilot line P11a to operate valve V11c and also along line P11b. Operation of valve V11c allows air to pass to piston and cylinder arrangements C11 and C12 whereby in sequence the plate 142 urges the shoe against the foot 166 and the side clamps clamp the shoe. If the shoe is not correctly positioned, release valve V11e is manually operated

which allows air from main line P1m to pass along line P11k to return valve V11c and thus piston and cylinder arrangements C11, C12 to their positions shown in FIG. 9.

As above described, if the template orientation matches the shoe place in the shoe support, air under pressure passes through the shuttle valve V22a along pilot line P22a and switches valve V22b from its position shown in FIG. 9 to allow air under pressure from line P11b along line P22b to operate valve V11d.

With the shoe correctly positioned valve V11d is operated manually to allow air through pilot line P11d to operate valve V11f. Operation of valve V11f allows air along pilot line P11f to hydraulic valve VH1, operation of which is effective to initiate movement of the table 32 to carry the shoe supported thereon beneath the brushes 38. At this time valves V13a, V13b, V14a and V14b are in positions shown in FIG. 9 with air passing therethrough from regulator valve RV13a along lines P13a, P14a, respectively, to piston and cylinder arrangements C13, C14. Thus the brushes 38 are being supported by the arrangements C13, C14 at a pressure regulated by valve RV13a. When the shoe support moves from the loading/unloading position, valve V30b is released and moves to the right from the position shown in FIG. 10. This causes valve V11h also to move from its position shown in FIG. 9 whereby air is admitted to accumulator A1.

As the shoe support 36 moves beneath the brushes progressively, firstly the cam member 358 operates valve V13a whereby air from regulator valve RV13b is then caused to pass through valve V13b and line P13a to piston and cylinder arrangement C13, and thereafter valve V14a is similarly operated, air from regulator valve RV13b passing through valve V14b and line P14a to piston and cylinder arrangement C14. After further movement of the shoe support valve V13b is operated by the cam member 372 whereby air under high pressure is passed from regulator valve RV13c along line P13a to piston and cylinder arrangement C13 to lift the brush out of engagement with the shoe bottom. Similarly the second brush is lifted when the cam member 372 operates valve V14b and air thus passes along line P14b to piston and cylinder arrangement C14.

Operation of valve V14b is also effective to allow air to pass along pilot line P14c to operate valve V14c whereby air from main line P1m passes along pilot lines P14d, P14e, P14f to operate valves V14d, V14e and to return valve V11f to its position shown in FIG. 10. Operation of valve V11f as aforesaid is effective to admit air under pressure along line P11h, through valve V30b (which is held by the cam in its position as shown in FIG. 10 when the shoe support is in its loading/unloading position but is moved to the right by a spring when the shoe support moves out of said position), and along line P30b, through the shuttle valve ShV1, to the left-hand end of valve VH1. Valve VH1 is thus moved to the right, line P11f being exhausted upon switching valve V11f, and fluid under pressure is admitted to the left-hand end of piston and cylinder arrangement H1, thus to return the shoe support to its loading/unloading position. Also, as set out above, operation of valve V14b is effective to set up a circuit for the reversal of the template through 180°

The movement of the shoe support is thus reversed, its return movement being at a faster speed than its movement to carry the shoe S beneath the brushes as aforesaid, which latter movement is restrained by means of a restrictor arrangement generally designated RH1 connected to the left hand end of piston and cylinder arrangement H1.

As the shoe support is caused to return, the cam member 702 engages and actuates valve V30a to move it into its position shown in FIG. 10, from which position it is urged by a spring when not engaged by the cam member. Actuating valve V30a admits air under pressure from main line P1m along a line P30a to switch hydraulic valve VH2 from its shut-off position (shown in FIG. 10) to a restricted bleed-off position, so that fluid under pressure from hydraulic pump P is bled off and thus causes deceleration of the shoe support to be effected.

As the shoe support continues its return movement the upper cam member 700 engages and actuates valve V30b which is thus returned to its position shown in FIG. 10. Actuating valve V30b causes line P30b to be exhausted and hydraulic valve VH1 is thus returned to its central shut-off position so that movement of the shoe support is arrested. Actuation of valve V30b is also effective to return valves V13a, V13b, V14a, V14b, V14d, V14e and V11h to their positions shown in FIGS. 9 and 10. Operation of valve V11h allows air from accumulator A1 through pilot line P11k to return valve V11c to its position shown in FIG. 9 and thus to exhaust piston and cylinder arrangements C11, C12 to release the shoe. Operation of the other four valves is effective to allow air from regulator valve RV13a once more to pass to piston and cylinder arrangements C13, C14. Return of valve V14b to its position shown is also effective to cause valve V23b to be returned to its position shown in FIG. 9 so that the flow of air to piston and cylinder arrangement C15 is reversed. The template T is thus rotated through 180°. In addition the indicator 394 is switched to indicate that the opposite shoe is next to be operated upon.

If at any time in a cycle of operation of the illustrative apparatus the operator wishes to curtail the operation, he operates manual return valve V14f which causes valves V14d, V14e and V11f to be operated to return the shoe support to its loading/unloading position and also ensures, through valves V14d and V14e, that air from regulator valve RV13c is admitted to piston and cylinder arrangements C13, C14 and that the brushes 38 are thus raised out of engagement with the shoe bottom.

It will thus be appreciated that using the illustrative apparatus, left and right shoes are operated upon alternately, such operation being controlled by means of the template, which is selected according to the style and size of the shoe being operated upon. It will also be appreciated that, when the illustrative apparatus is in use and a newly selected template has been positioned as aforesaid in the template supporting means, in the first subsequent cycle of operation the shoe support will start from a left-hand position (viewing FIG. 1) as far remote from the brushes as the construction of the illustrative apparatus allows, but thereafter is caused to return under the operation of valve V30b to a loading/unloading position determined, through the locating ar-

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rangement aforementioned, according to the style and size of a template T supported by the template supporting means. Since the locating arrangement is also effective to locate the toe end portion supporting device of the shoe support, it will be appreciated that the loading/unloading position, at which the return movement of the shoe support is arrested, is set according to the position in the shoe support of the end portion, nearer to the brush supporting means, (viz. the toe end portion) of a shoe S supported by the shoe support.

Having thus described our invention what we claim as new and desire to secure by Letters Patent of the United States is:

1. Apparatus for use in operating on marginal portions of shoe bottoms comprising a shoe support, tool supporting means, means for effecting relative movement between the shoe support and the tool supporting means in one direction whereby, when the apparatus is in use, a tool supported by the tool supporting means is caused to operate progressively along a marginal portion of a shoe supported by the shoe support, and thereafter in an opposite direction, and means whereby such relative movement in the opposite direction can be arrested, the arresting means being adjustable to operate relative to the position of the end portion of the shoe nearer to the tool supporting means.

2. Apparatus according to claim 1 further including means for supporting a template, and sensing means for cooperating with the template supported by a template supporting means, the template supporting means cooperative with the shoe support and being operatively connected to the tool supporting means to control the tool supporting means so that a tool supported thereby follows a path corresponding to the shape of a marginal portion of the bottom of a shoe supported by the shoe support and operates progressively along said marginal portion.

3. Apparatus according to claim 2 wherein the arresting means comprises a switch arrangement and an actuator arrangement therefor which are positioned relative to one another according to the size and style of a template supported by the template supporting means.

4. Apparatus according to claim 3 wherein the actuator arrangement comprises an actuator member positioned on the shoe support in a position determined by the style and size of a template supported by the template supporting means.

5. Apparatus according to claim 3 wherein the switch arrangement comprises a control valve by which is controlled the means for effecting relative movement between the shoe support and the tool supporting means.

6. Apparatus according to claim 2 including decelerating means whereby relative movement in the opposite direction between the shoe support and the tool supporting means can be decelerated prior to being arrested.

7. Apparatus according to claim 6 wherein the decelerating means comprises a switch arrangement

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and an actuator arrangement therefor which are positioned relative to one another according to the style and size of a template supported by the template supporting means.

8. Apparatus according to claim 7 wherein the actuator arrangement comprises an actuator member positioned on the shoe support in a position determined by the size and style of a template supported by the template supporting means.

9. Apparatus according to claim 6 wherein the means for effecting relative movement between the shoe support and the tool supporting means is fluid pressure operated and a switch arrangement for the decelerating means is included which comprises a restrictor valve by which the flow of pressure fluid to said fluid pressure operated means is restricted.

10. Apparatus according to claim 2 wherein the shoe support includes two devices by which opposite end portions of a shoe can be supported, and the one of said devices by which the end portion, nearer to the tool supporting means of a shoe is supported, is mounted for adjusting movement relative to the other of said devices, said one device being positioned in the shoe support according to the size and style of a template supported by the template supporting means.

11. Apparatus according to claim 10 wherein the shoe support is movable relative to the tool supporting means along a path extending generally lengthwise of a shoe supported by the shoe support.

12. Apparatus suitable for use in operating on marginal portions of shoe bottoms comprising a shoe support, tool supporting means, means cooperating with the shoe support for supporting a template selected according to the style of shoe to be operated upon, sensing means operatively connected to the tool supporting means for cooperating with a template supported by the template supporting means, and means for effecting movement of the shoe support and the cooperating template supporting means relative to the tool supporting means, relative to the sensing means, in one direction, along a path extending generally lengthwise of a shoe supported by the shoe support whereby a tool supported by the tool supporting means operates progressively along a marginal portion of the bottom of a shoe supported by the shoe support, and for thereafter moving said shoe support in an opposite direction, and means for arresting movement of the shoe support moving in said opposite direction at a point in the path selectable according to the size and style of a template supported by the template supporting means.

13. Apparatus according to claim 2 wherein a pair of tools supported by the tool supporting means are caused to operate progressively along opposite marginal portions of the bottom of a shoe supported by the shoe support.

14. Apparatus according to claim 13 wherein the tools are rotary wire brushes, supported by the tool supporting means.

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