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TITLE OF THE INVENTION  
CONICAL PISTON SOLIDS DISCHARGE  
AND PUMPING CENTRIFUGAL SEPARATOR

5                   CROSS REFERENCE TO RELATED APPLICATIONS

        This patent application is a continuation-in-part of U.S. Application No. 10/823,844 filed on April 14, 2004 entitled, CONICAL PISTON SOLIDS DISCHARGE CENTRIFUGAL SEPARATOR, the whole of which is hereby incorporated by reference herein.

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BACKGROUND OF THE INVENTION

        The present invention generally relates to centrifuges and in particular to centrifuges enabling the automatic discharge and pumping of solids that accumulate during separation.

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        Many different types of centrifugal separators are known for separating heterogeneous mixtures into components based on specific gravity. A heterogeneous mixture, which may also be referred to as feed material or feed liquid, is injected into a rotating bowl of the separator. The bowl rotates at high speeds and forces particles of the mixture, having a higher specific gravity, to separate from the liquid by sedimentation. As a result, a dense solids cake compresses tightly against the surface of the bowl, and the clarified liquid, or "centrate", forms radially inward from the solids cake. The bowl may rotate at speeds sufficient to produce forces 20,000 times greater than gravity to separate the solids from the centrate.

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        The solids accumulate along the wall of the bowl, and the centrate is drained off. Once it is determined that a desired amount of the solids has been accumulated, the separator is placed in a discharge mode in which the accumulated solids are removed from the separator. In a typical configuration, an internal scraper is engaged to scrape the solids from the walls of the separator bowl.

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Prior separators have shortcomings when operating with particular kinds of materials. For example, many separators may not be capable of completely discharging residual solids that are sticky, which can result in poor yield. This can be especially  
5 problematic for high-value materials such as are encountered in pharmaceutical processes. Additionally, many separators subject the feed material to very high shear forces when accelerating the feed liquid to the rotational speed of the bowl, which can damage sensitive materials such as pharmaceuticals or biological  
10 substances that include intact cells. Other existing separators do not provide a convenient means by which to handle and recover these sensitive materials.

It would be desirable to have a centrifugal separator that can be effectively used with materials of the type described,  
15 namely those that result in sticky accumulated solids and those that are sensitive to shear forces generated during the centrifuge process. It would also be useful to have a separator that can easily recover such solids without the possibility of external contamination or additional operator handling.

#### BRIEF SUMMARY OF THE INVENTION

In accordance with the present invention, a centrifugal separator is disclosed that performs well with sticky solids and that exhibits low-shear acceleration of feed liquid, making the  
25 separator particularly useful for sensitive materials such as pharmaceutical and biological materials. The separator is also useful for recovering these sensitive materials without further handling.

The separator includes a cylindrical bowl having a conical  
30 lower end with an opening through which feed liquid is injected during a feed mode of operation. As the bowl rotates at a high speed, the injected feed liquid encounters the sloped surface of the conical lower end of the bowl first. Rotational acceleration

forces are imparted relatively gradually as the liquid continues its movement radially outward. The feed liquid is ultimately separated into centrate and solids, the solids accumulating along the inner surface of the bowl.

5 The separator further includes a piston assembly including a conical piston coupled to a piston actuator, with the piston being disposed within the bowl in tight-fitting relationship with the inner surface thereof. In a solids discharge mode of operation, the piston actuator urges the piston axially downward to force the  
10 accumulated solids from the bowl via the opening in the conical lower end of the bowl. The conical shape promotes relatively complete discharge of the solids.

In the disclosed separator, the piston is held in an uppermost position during the feed mode of operation by hydraulic  
15 pressure from the feed liquid. The piston includes a centrate valve that is urged open during the feed mode of operation to permit the centrate to flow out of the bowl and into a passage leading to a centrate discharge port. As the piston is urged downward during the solids discharge mode of operation, the  
20 centrate valve automatically closes, preventing the accumulated solids from passing into the centrate passage.

In another embodiment, the separator includes a divert assembly including a solids divert valve movably located below a rotatable residual divert valve, when the residual divert valve is  
25 at the opening in the conical lower end of the bowl. In a solids pumping mode of operation, a residual divert valve actuator rotates the residual divert valve such that the solids divert valve can be urged upward into communication with the opening in the bowl by a solids divert piston. The conical piston is then  
30 urged axially downward by its actuator to push or "pump" the accumulated solids from the bowl into a passage leading to a solids outlet port.

The disclosed separator also includes a two-part piston shaft having a connected position and a disconnected position. When the piston shaft is in the disconnected position, the piston is permitted to be forced upwardly and to rotate with the bowl. When the piston shaft is in the connected position, the piston can be pushed and pulled axially by the piston actuator, thus facilitating the solids discharge and pumping modes of operation.

Other aspects, features, and advantages of the present invention will be apparent from the Detailed Description that follows.

#### BRIEF DESCRIPTION OF THE SEVERAL VIEWS OF THE DRAWINGS

Figure 1 is a section view of a first embodiment of a centrifuge in accordance with the present invention;

Figure 2 is a detailed section view of an upper portion of the centrifuge of Figure 1;

Figure 3 is a partial section view of a central region of the centrifuge of Figure 1 illustrating operation in feed mode;

Figure 4 is a detailed section view of the central region of the centrifuge of Figure 1 illustrating operation in solids discharge mode;

Figure 5 is a detailed section view of the upper portion of the centrifuge of Figure 1 when a piston shaft is disconnected to permit rotation of the bowl;

Figure 6 is a detailed section view of the upper portion of the centrifuge of Figure 1 when the piston shaft is connected to move a piston axially within the bowl;

Figure 7 is a section view of a second embodiment of the centrifuge in accordance with the present invention;

Figure 8 is a section view of the centrifuge of Figure 7 illustrating operation in feed mode;

Figure 9 is a section view of the centrifuge of Figure 7 illustrating operation when centrate drains from the bowl;

Figure 10 is a section view of the centrifuge of Figure 7 illustrating operation in pumping mode; and

Figure 11 is a detailed section view of a lower end region of the centrifuge of Figure 7 when a solids passage is cleaned.

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#### DETAILED DESCRIPTION OF THE INVENTION

Figure 1 shows a centrifugal separator in vertical section, with a middle portion removed so as to illustrate a horizontal section as well. The centrifugal separator includes a cylindrical separator bowl 10 mounted in a central region 11 of a separator housing 13. The separator bowl 10 is preferably a cylindrical type bowl having a relatively small diameter D and a length L such that the ratio of L/D is approximately 5/1 or greater. The separator includes a piston assembly consisting of a piston 12 connected to a piston shaft 14. As shown, the piston 12 has a conical shape that matches the shape of a conical feed cone 17 of the bowl 10. The feed cone 17 acts as a rotational accelerator of the feed liquid during a feed mode of operation of the separator.

A variable speed drive motor 16 is connected by a drive belt 5 to a drive pulley 18 of a spherically mounted bearing and spindle assembly 20 located at a collar-like extension 21 of the upper end of the separator housing 13. The drive motor 16 is controllably operated to rotate the separator bowl 10 at desired speeds for separating the feed liquid.

A piston shaft coupling lock cylinder 22 is mounted in a crosshead 24 of a piston actuator which includes two piston actuator plungers 26 mounted in respective piston actuator cylinders 28. Each piston actuator plunger 26 is operatively connected to the piston shaft 14 via the crosshead 24 for raising and lowering the piston 12 within the separator bowl 10 in response to compressed air or hydraulic fluid introduced at piston actuator ports 29. As described in greater detail below, the piston shaft 14 includes two parts that are selectively connected

together or disconnected depending on the operating mode, such that the piston 12 is permitted to rotate with the bowl 10 when the parts are disconnected, and can be moved axially within the bowl 10 when the parts are connected.

5 Also shown in Figure 1 are a centrate case 30, a centrate outlet port 32 and a centrate valve 34, all of which are involved in removing the centrate, or clarified liquid, from the centrifugal separator during operation, as described in more detail below. A solids valve 38 is mounted in a lower end region  
10 39 of the separator housing 13, below an inward-facing flange 41. The solids valve 38 incorporates both a feed liquid passage 40 in communication with a feed liquid port 42, as well as a residual liquid drain passage 44 in communication with a residual liquid drain port 46. A solids valve seal 48 is disposed on a lower  
15 surface of the flange 41. The solids valve 38 is shown in a closed position maintained during the feed mode of operation of the separator. The solids valve 38 can be rotated along axis 6 to an open position such that accumulated solids can be discharged through the lower end of the separator during the solids discharge  
20 mode of operation.

Figure 2 shows the upper portion 19 of the separator in greater detail. The two-part piston shaft 14 includes an upper piston shaft 50 with a coupling portion 52, and a lower piston shaft 54. The manner in which the upper and lower piston shafts  
25 50 and 54 engage each other is described below. Figure 2 also shows piston seals 56 that seal the interface between the piston 12 and the inner surface of the bowl 13. The seals 56 are of the type commonly referred to as O-ring loaded lip seals, and are made of a TEFLON-containing (E. I. du Pont de Nemours and Company, 1007  
30 Market Street, Wilmington, Delaware 19898) elastomeric material. Centrate seals 57 of similar construction seal the interface between the centrate valve 34 and the upper part of the piston 12 when the centrate valve 34 is closed.



In Figure 2, the centrate valve 34 is shown in the open position, which results from the downward pushing action of pins 58 extending from a hub 60 through openings 62 in the upper part of the piston 12. With the centrate valve 34 in this open position, centrate can flow through small grooves 64. The centrate valve 34 is open only when the piston 12 is at its uppermost position against the hub 60. As the piston is pushed downward away from the pins 58 by the piston actuator, springs 66 urge the centrate valve slightly upward to a closed position, which is maintained throughout the solids discharge process.

Figure 3 illustrates operation of the separator during a feed mode of operation, during which the bowl 10 and piston 12 are rotating at high speed. Solids-bearing feed liquid flows in a path 68 up the inner surface of the conical feed cone 17. Under the separation forces generated by high-speed rotation of the bowl 10, the feed liquid is separated into accumulated solids 70 and a relatively solids-free centrate 72. Hydraulic pressure from the centrate 72 holds the piston 12 upward against the hub 60 of the bowl, maintaining the centrate valve 34 in the open position. At the inner surface of the centrate 72, it flows through the grooves 64 of the centrate valve 34 and continues upward along a discharge path until exiting the bowl at a centrate discharge opening 74.

Figure 4 illustrates operation of the separator during a solids discharge mode of operation. Figure 4 is split lengthwise to show two separate positions of the piston 12. On the left, the piston 12 is partway through its downward travel, and on the right, the piston 12 is at its lowermost point at the completion of the discharge operation, with its conical outer surface resting against the inner surface of the conical feed cone 17. It will be observed that the centrate valve 34 is closed, under the upward urging force of the springs 66. As the piston 12 travels downward, the accumulated solids 70 are pressed out of an opening 76 at the bottom of the bowl 10. The conical outer surface of the

piston 12 and the inner surface of the conical feed cone 17 are machined for a precise fit, so that the squeezing action of these two surfaces can efficiently remove as much of the solids 70 as possible. Any solids remaining after the discharge process are removed by clean-in-place processes after the piston 12 is returned to its uppermost position by the piston actuator.

Figures 5 and 6 illustrate the configuration and operation of the mechanical coupling between the piston 12 and the crosshead 24. The upper piston shaft 50 extends from the underside of the crosshead 24 and moves with it in response to actuation by the piston plungers. Figure 5 shows the upper piston shaft in a disconnected position in which it is withdrawn from an upper hollow portion 78 of the lower piston shaft 54. It will be observed that the hollow portion 78 includes a slightly wider chamber 79 whose use is explained below. The upper piston shaft 50 is hollow along its entire length, and a coupling lock draw bar 80 is disposed therein. At its lower end, the upper piston shaft 50 includes a plurality of flexible fingers 81 whose function is explained below.

At the upper end, the coupling lock draw bar 80 is mechanically connected to a coupling lock piston 82 located within the coupling lock cylinder 22, such as by a horizontal pin 84 as shown. The coupling lock piston 82 is biased to a downward position by a spring 83. At its lower end 86, the coupling lock draw bar 80 has a flared shape for use in locking the upper and lower piston shafts 50, 54 together as described below.

When the piston shaft is in the disconnected position shown in Figure 5, the piston 12 is free to move in response to forces other than those generated by the piston actuator. In particular, the piston 12 is held upwardly by hydraulic forces and rotates with the bowl 10 during the feed mode of operation, as described above. It should be noted that at the very beginning of the feed mode of operation, before sufficient hydraulic pressure is

present, the piston is held at substantially its uppermost position by frictional forces between the seals 56 and the inner wall of the bowl 10 (Fig. 3). As hydraulic pressure builds, the piston is then pushed upward firmly enough to open the centrate valve 34.

Figure 6 shows the upper piston shaft in a connected position in which it is inserted into the upper hollow portion 78 (Fig. 5) of the lower piston shaft 54. The insertion typically occurs just prior to the solids discharge operating mode, when the piston 12 is located at its uppermost position within the bowl 10, as the piston actuator lowers the upper piston shaft 50 into the lower piston shaft 54. Then, hydraulic or pneumatic pressure 88 is provided to urge the coupling lock piston 82 upwardly, which in turn urges the coupling lock draw bar 80 upwardly with respect to the upper piston shaft 50. The flanged lower portion 86 of the coupling lock draw bar 80 pushes against the fingers 81 and urges them against the walls of the chamber 79, locking the upper and lower piston shafts 50, 54 together. In this connected configuration, the axial forces generated by the piston actuator cause the piston 12 to move axially. The connected configuration is maintained throughout the solids discharge mode of operation in which the piston 12 is moved downwardly to discharge the accumulated solids. The piston 12 is drawn to its uppermost position when the discharging of solids is complete, and the upper piston shaft 50 is then disconnected from the lower piston shaft 54 in preparation for the next cycle of feed mode operation. As mentioned above, the piston 12 remains in this position due to the frictional forces between the seals 56 and the inner wall of the bowl 10 (Fig. 3).

Figure 7 illustrates another embodiment of the separator in vertical section, with a middle portion removed like Figure 1 so as to illustrate a horizontal section. In this embodiment, a solids divert valve 90 is movably located in the lower end region

39 of the separator housing 13, below a lower surface of a rotatable residual divert valve 92. The lower surface of the residual divert valve can have a feature that extends partially within the solids divert valve 90. The residual divert valve  
5 located at the opening 76 in the bottom of the bowl 10 is shown in a closed position, which is maintained during the feed mode. When closed, the valve 92 defines the feed liquid passage 94 in communication with the feed liquid port 96, as well as the residual drain passage 98 in communication with the residual  
10 liquid drain port 100. The valve 92 can be rotated from its closed position about axis 6 such that the solids divert valve 90 can be urged upward into communication with the opening 76 to the bowl, as described below for operation in a solids pumping mode. As shown, the lower end region 39 generally characterizes this  
15 embodiment with the central region 11 and the upper portion 19 of the separator being similar to the embodiment described above (Figs. 1-6).

Also shown in Figure 7 is a solids passage 104 disposed axially within a solids divert piston 102 and extending beyond the  
20 piston 102 at a lowermost end to incorporate a solids outlet port 106. The passage, piston and port 104, 102, 106 are each involved in removing the solids from the centrifugal separator during the solids pumping mode. The solids pumping mode recovers sensitive solids, such as, for example, intact cells, and can pass these  
25 solids onto another process or a storage vessel without further handling. Without the solids being handled by an operator, they are less likely to be damaged. A cleaning passage 108 is also disposed within the solids divert piston 102 parallel to the solids passage 104, extending beyond the piston at a lowermost end  
30 to incorporate a cleaning port 111. At an uppermost end, the cleaning passage 108 is in communication with the solids passage 104. The cleaning passage and port 108, 111 together aid in the recovery of any solids remaining in the passage 104 following the

solids pumping mode, as well as in cleaning or sterilizing the separator.

Figure 8 illustrates the present embodiment during the feed mode of operation, during which the bowl 10 and the piston 12 are rotating together at high speed. As shown, the feed mode is generally similar to that described in the previous embodiment (Fig. 3), wherein the solids-bearing feed liquid is injected into the bowl and flows in a path 64 up the inner surface of the conical feed cone 17. The upper piston shaft 50 is in a disconnected position in which it is withdrawn from the upper hollow portion 78. The piston is held at its uppermost position by hydraulic pressure from the centrate such that the piston is urged against the hub 60 of the bowl, maintaining the centrate valve 34 in the open position. Under the separation forces generated by high-speed rotation of the bowl, the feed liquid is separated into accumulated solids 70 and the relatively solids-free centrate 72. The centrate continues upward along the discharge path, through the centrate valve 34 and exits the bowl at the centrate discharge opening 74. The discharge opening 74 leads into the centrate case 30 and the outlet port 32, where the centrate eventually exits the separator.

During the feed mode, the solids divert valve 90 is held upwardly against a lower surface of the residual divert valve 92 in gas-tight agreement. Seals made of TEFLON-containing (E. I. du Pont de Nemours and Company, 1007 Market Street, Wilmington, Delaware 19898) elastomeric materials can be disposed on the divert valves 90, 92 to seal the interface between them. The solids divert valve is urged upward by the solids divert piston 102 on which the valve 90 is disposed at an uppermost end in communication with the solids passage 104 of the piston 102. As shown, compressed air or hydraulic fluid introduced at an actuator port 112 acts on a lower surface of an annular flange 110 disposed about the divert piston to urge the piston upward. The divert

piston 102 moves axially upward and downward in response to pneumatic or hydraulic pressure. A control port 113 may also be provided in the lower end region 39 of the separator to aid the actuator port in its movement of the divert piston.

5 Also shown in Figure 8 is the residual divert valve 92 in a closed position located at the opening 76 in the bottom of the bowl 10. The valve 92 defines the feed liquid passage 94 in communication with the feed liquid port 96 such that the feed liquid can be injected into the bowl along the path 64.  
10 Operatively coupled to the valve 92 is a residual divert valve actuator 114. The actuator, which can be a hydraulic or pneumatic cylinder, rotates the valve 92 from its closed position about axis 6.

Figure 9 illustrates the separator with the residual divert  
15 valve 92 closed to permit the centrate 72 to drain out of the bowl 10 and into the residual liquid drain passage 98. The drain passage 98 leads into the residual liquid drain port 100, where the centrate eventually drains from the separator. In both embodiments of the separator, the centrate 72 is typically drained  
20 from the separator by gravity after the feed mode is completed and the high-speed rotational separation is performed. Although the bowl 10 and the piston 12 are no longer rotating, accumulated solids 70 remain compressed tightly against the inner surface of the bowl. As shown, the upper piston shaft 50 is in a  
25 disconnected position in which it is withdrawn from the upper hollow portion 78. The piston 12 is held substantially at its uppermost position by frictional forces between the seals 56 and the inner wall of the bowl (Fig. 3). Each of the separator embodiments may allow centrate to drain from the bowl and could be  
30 performed, for example, by the configuration shown in Figure 9.

After solids separation and the residual centrate 72 has drained from the bowl 10, the residual divert valve 92 is rotated and opened to enable the pumping of the accumulated solids 70 from

the separator. Figure 10 illustrates the present embodiment of the separator operating in the solids pumping mode. Figure 10 is split lengthwise like Figure 4 to show two separate positions of the piston 12. On the left, the piston is partway through its downward travel, and on the right, the piston is at its lowermost point at the completion of the pumping operation, with its conical outer surface resting against the inner surface of the conical feed cone 17. As shown, the centrare valve 34 is closed, under the upward urging force of the springs 66 and by the interaction between the accumulated solids and the conical outer surface of the piston 12 during its downward travel. With the piston 12 traveling downward, the accumulated solids 70 are pressed out of the opening 76 at the bottom of the bowl 10. Similar to the embodiment described above, the conical outer surface of the piston and the inner surface of the conical feed cone are machined for precise fit to efficiently remove as much of the accumulated solids from the bowl as possible.

Also shown in Figure 10 is the piston shaft coupling lock cylinder 22 mounted in the crosshead 24 of the piston actuator with its actuator plungers 26. As described above, the actuator plungers 26 are operatively connected to the piston shaft 14 via the crosshead for axially pulling or pushing the piston 12 within the bowl 10 in response to compressed air or hydraulic pressure introduced at the piston actuator ports 29. Although not shown, the upper piston shaft 50 is inserted into the upper hollow portion 78 of the lower piston shaft 54 (Fig. 6) before the solids pumping mode begins, and when the piston is located at its uppermost position within the bowl 10. Pneumatic or hydraulic pressure 88 is then introduced to urge the coupling lock piston 82 upward, which in turn urges the coupling lock draw bar 80 upward with respect to the upper piston shaft. The flanged lower portion 86 of the coupling lock draw bar 80 pushes against the fingers 81 and urges them against the walls of the chamber 79, locking the

upper and lower piston shafts 50, 54 together (Fig. 6). When connected, the actuator plungers 26 can push or pull the piston within the bowl. The solids pumping mode begins in the upper portion 19 of the separator by the piston moving axially downward to push the accumulated solids 70 from the bowl.

At the lower end region 39 of the separator, the solids pumping mode begins with the solids divert piston 102 being lowered by reduction of the compressed air or hydraulic fluid pressure previously applied at actuator port 112. The residual divert valve 92 is then rotated from its closed position by the residual divert valve actuator 114. The valve actuator 114 rotates the residual divert valve about axis 6 in response to pneumatic or hydraulic pressure. The residual divert valve is preferably rotated 90° from its closed position. The solids divert valve 90 can then be urged upward by the solids divert piston 102. As shown, compressed air or hydraulic fluid introduced at the actuator port 112 acts on the annular flange 110 of the piston 112 to urge it axially upward such that the solids divert valve is held in gas-tight communication with the opening 76 at the bottom of the bowl 10. The interface of the valve 90 and the bowl opening 76 can also be sealed by TEFLON-containing (E. I. du Pont de Nemours and Company, 1007 Market Street, Wilmington, Delaware 19898) elastomeric seals disposed therebetween such that the pumped solids cannot be contaminated.

Operating the separator in the solids pumping mode is generally similar to the discharge mode described above (Fig. 4). Although in the present embodiment, solids pushed through the opening 76 in the bottom of the bowl 10 pass into the solids passage 104 disposed partially within the solids divert piston 102 below the solids divert valve 90. The solids passage 104 extends beyond the lowermost end of piston 102 leading into the solids outlet port 106, where the pumped solids eventually exit and are recovered from the separator. The outlet port 106 can pass the



pumped solids onto another process or a storage vessel without further handling by an operator or the opportunity for contamination.

As described, the solids pumping mode of operation is completed as the piston 12 reaches the lowermost point of its downward stroke and rests against the inner surface of the conical feed cone 17. The piston is returned to its uppermost position by actuation of the piston plungers 29, when the pumping of solids is complete. The solids divert valve 90 is also drawn downward by movement of the solids divert piston 102 such that the residual divert valve 92 can be rotated to its closed position about rotational axis 6 in response to compressed air or hydraulic fluid acting on the residual divert valve actuator 114. The solids divert valve 90 is then urged upward against the lower surface of the residual divert valve 92. At the upper portion 19 of the separator, the upper piston shaft 50 is then disconnected from the lower piston shaft 54 in preparation for the next cycle of feed mode operation (Fig. 5).

Figure 11 illustrates the lower end region of the separator in greater detail with the residual divert valve 92 returned to its closed position. The solids divert valve 90 is held upward against the lower surface of the residual divert valve by the solids divert piston 102. As shown, compressed air or hydraulic fluid introduced at an actuator port 112 acts on a lower surface of an annular flange 110 disposed about the divert piston to urge it upward. Although the solids pumping mode is completed, solids can remain in the solids passage 104 of the piston 102. To remove the remaining solids, compressed air or hydraulic fluid is introduced to the cleaning port 111 of the cleaning passage 108.

The cleaning passage and port 108, 111 extend beyond the lowermost end of the piston 102, with the cleaning passage partially disposed within the piston. The cleaning passage is also in communication at its uppermost end with the solids passage

104 of the piston 102. This communication permits compressed air or hydraulic fluid introduced at the cleaning port to pass through the cleaning passage 108 and into the solids passage 104. The compressed air or hydraulic fluid pushes the remaining solids in the passage 104 toward the solids outlet port 106. As shown, the solids passage 104 is in communication with the outlet port 106 such that any remaining solids in the passage can exit the separator. The outlet port 106 can pass the recovered solids onto another process or a storage vessel without further handling.

10 The cleaning passage and port 108, 111 can also be used to clean or sterilize the solids passage 104 and outlet port 106. In both embodiments of the separator, such clean-in-place or sterilize-in-place processes are convenient for preparing the centrifugal separator for the next cycle of operation and may be performed, for example, by the configuration shown in Figure 11. These processes also reduce the likelihood of operator contamination.

20 While the present invention has been described in conjunction with a preferred embodiment, one of ordinary skill in the art, after reading the foregoing specification, will be able to effect various changes, substitutions of equivalents and other alterations to the compositions, articles, methods and apparatuses set forth herein. Furthermore, the embodiments described above may each include or incorporate any of the variations of all other embodiments. It is therefore intended that the protection granted by Letter Patent hereon be limited only by the definitions contained in the appended claims and equivalents thereof.

## CLAIMS

What is claimed is:

1. A divert assembly for a centrifugal separator, comprising:

5 a first valve member proximate to an opening of a separator bowl, the first valve member operatively coupleable to a valve actuator for rotating the first valve member about a rotational axis;

10 a second valve member cooperating with a lower surface of the first valve member when the first valve member is in a closed position; and

a valve piston having an uppermost end at which the second valve member is proximately disposed, the valve piston operative to move the second valve member with respect to the bowl.

15 2. The divert assembly for a centrifugal separator of claim 1, wherein during a solids pumping mode of operation the valve piston moves the second valve member upward along a vertical axis to cooperate with the opening in the bowl.

20 3. The divert assembly for a centrifugal separator of claim 1, wherein during a feed mode of operation the first valve member is in the closed position permitting feed liquid to be injected into the bowl.

25 4. The divert assembly for a centrifugal separator of claim 3, wherein the first valve member defines a feed passage, the feed passage cooperating with the opening in the bowl during the feed mode of operation to permit feed liquid to be injected into the bowl.

30 5. The divert assembly for a centrifugal separator of claim 1, wherein the first valve member defines a drain passage, the drain

passage operative to permit centrate to drain from the opening in the bowl when the first valve member is in the closed position.

6. The divert assembly for a centrifugal separator of claim 5, wherein the opening in the bowl and the drain passage are configurable to enable centrate to drain by gravity from the bowl into the drain passage.

7. The divert assembly for a centrifugal separator of claim 1, wherein a first passage is partially disposed within the valve piston, the first passage cooperating with the second valve member at the uppermost end of the valve piston.

8. The divert assembly for a centrifugal separator of claim 7, wherein the first passage extends beyond a lowermost end of the valve piston.

9. The divert assembly for a centrifugal separator of claim 7, wherein the opening in the bowl and the first passage are configurable to enable solids from the bowl to pass through the first passage during a solids pumping mode of operation.

10. The divert assembly for a centrifugal separator of claim 7, wherein the first passage cooperates with a second passage partially disposed within the valve piston.

11. The divert assembly for a centrifugal separator of claim 10, wherein the second passage extends beyond a lowermost end of the valve piston.

12. The divert assembly for a centrifugal separator of claim 11, wherein compressed air or hydraulic fluid introduced through a

port for the second passage enters the first passage to contact solids therein.

13. The divert assembly for a centrifugal separator of claim 1,  
5 wherein an annular flange is disposed about the valve piston.

14. The divert assembly for a centrifugal separator of claim 13,  
wherein the valve piston moves in response to compressed air or  
hydraulic fluid contacting the annular flange.

10 15. A divert assembly for a centrifugal separator, comprising:  
a first valve assembly, wherein the first valve assembly  
comprises

15 a first valve member, the first valve member proximate  
to an opening of a separator bowl, and

an actuator operatively coupleable to the first valve  
member for rotating the first valve member about a rotational  
axis; and

20 a second valve assembly, wherein the second valve assembly  
comprises

a second valve member, the second valve member  
cooperating with the first valve member of the first valve  
assembly when the first valve member is in a closed position, and

25 a valve piston, the valve piston having an uppermost  
end at which the second valve member is proximately disposed,  
wherein the valve piston is operative to move the second valve  
member with respect to the bowl.

16. The divert assembly for a centrifugal separator of claim 15,  
30 wherein during a solids pumping mode of operation the valve piston  
moves the second valve member upward along a vertical axis to  
cooperate with the opening in the bowl.

17. The divert assembly for a centrifugal separator of claim 15, wherein during a feed mode of operation the first valve member is in the closed position permitting feed liquid to be injected into the bowl.

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18. The divert assembly for a centrifugal separator of claim 17, wherein the first valve member defines a feed passage, the feed passage cooperating with the opening in the bowl during the feed mode of operation to permit feed liquid to be injected into the bowl.

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19. The divert assembly for a centrifugal separator of claim 15, wherein the first valve member defines a drain passage, the drain passage operative to permit centrate to drain from the opening in the bowl when the first valve member is in the closed position.

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20. The divert assembly for a centrifugal separator of claim 19, wherein the opening in the bowl and the drain passage are configurable to enable centrate to drain by gravity from the bowl into the drain passage.

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21. The divert assembly for a centrifugal separator of claim 15, wherein a first passage is partially disposed within the valve piston, the first passage cooperating with the second valve member at the uppermost end of the valve piston.

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22. The divert assembly for a centrifugal separator of claim 21, wherein the first passage extends beyond a lowermost end of the valve piston.

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23. The divert assembly for a centrifugal separator of claim 21, wherein the opening in the bowl and the first passage are

configurable to enable solids from the bowl to pass through the first passage during a solids pumping mode of operation.

24. The divert assembly for a centrifugal separator of claim 21,  
5 wherein the first passage cooperates with a second passage partially disposed within the valve piston.

25. The divert assembly for a centrifugal separator of claim 24,  
10 wherein the second passage extends beyond a lowermost end of the valve piston.

26. The divert assembly for a centrifugal separator of claim 25,  
wherein compressed air or hydraulic fluid introduced through a  
port for the second passage enters the first passage to contact  
15 solids therein.

27. The divert assembly for a centrifugal separator of claim 15,  
wherein an annular flange is disposed about the valve piston.

28. The divert assembly for a centrifugal separator of claim 27,  
20 wherein the valve piston moves in response to compressed air or hydraulic fluid contacting the annular flange.

29. A centrifugal separator, comprising:

25 a cylindrical bowl having a frustoconical lower end with an opening, the bowl being operative during a feed mode of operation to rotate at a high speed to separate the feed liquid into centrate and solids, the solids accumulating along an inner surface of the bowl;

30 a piston assembly including a conical piston coupled to a piston actuator, the piston being movably disposed against the inner surface of the bowl, the piston actuator operative to move

the piston along a vertical axis to force accumulated solids from the opening in the bowl; and

5 a first valve member, wherein the first valve member defines a drain passage, the drain passage operative to permit centrate to drain from the opening in the bowl when the first valve member is in the closed position.

10 30. The centrifugal separator of claim 29, wherein the opening in the bowl and the drain passage are configurable to enable centrate to drain by gravity from the bowl into the drain passage.

15 31. The centrifugal separator of claim 29, wherein the first valve member defines a feed passage, the feed passage cooperating with the opening in the bowl during the feed mode of operation to permit feed liquid to be injected into the bowl.

20 32. The centrifugal separator of claim 29, wherein the first valve member is operatively coupleable to a valve actuator for rotating the first valve member about a rotational axis.

33. A centrifugal separator, comprising:

25 a cylindrical bowl having a frustoconical lower end with an opening, the bowl being operative during a feed mode of operation to rotate at a high speed to separate the feed liquid into centrate and solids, the solids accumulating along an inner surface of the bowl;

30 a piston assembly including a conical piston coupled to a piston actuator, the piston being movably disposed against the inner surface of the bowl, the piston actuator operative to move the piston along a vertical axis to force accumulated solids from the opening in the bowl;

a first valve assembly, wherein the first valve assembly comprises



a first valve member, the first valve member proximate to an opening of a separator bowl, and

an actuator operatively coupleable to the first valve member for rotating the first valve member about a rotational axis; and

a second valve assembly, wherein the second valve assembly comprises

a second valve member, the second valve member cooperating with the first valve member of the first valve assembly when the first valve member is in a closed position, and

a valve piston, the valve piston having an uppermost end at which the second valve member is proximately disposed, wherein the valve piston is operative to move the second valve member with respect to the bowl.

34. The centrifugal separator of claim 33, wherein during a solids pumping mode of operation the valve piston moves the second valve member upward along a vertical axis to cooperate with the opening in the bowl.

35. The centrifugal separator of claim 33, wherein during a feed mode of operation the first valve member is in the closed position permitting feed liquid to be injected into the bowl.

36. The centrifugal separator of claim 35, wherein the first valve member defines a feed passage, the feed passage cooperating with the opening in the bowl during the feed mode of operation to permit feed liquid to be injected into the bowl.

37. The centrifugal separator of claim 33, wherein the first valve member defines a drain passage, the drain passage operative to permit centrate to drain from the opening in the bowl when the first valve member is in the closed position.

38. The centrifugal separator of claim 37, wherein the opening in the bowl and the drain passage are configurable to enable centrate to drain by gravity from the bowl into the drain passage.

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39. The centrifugal separator of claim 34, wherein the conical piston includes a centrate valve having an open position in the feed mode of operation and a closed position in the solids pumping mode of operation, the centrate valve being operative in the open position to permit the flow of the centrate from the bowl into a passage to a centrate discharge port of the separator, the centrate valve being operative in the closed position to block the passage of the accumulated solids from the bowl into the passage to the centrate discharge port.

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40. The centrifugal separator of claim 39, wherein the centrate valve is an annular member located at substantially the midsection of the conical piston.

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41. The centrifugal separator of claim 39, wherein the conical piston is configured to bias the centrate valve in the closed position, the bowl includes a hub against which the conical piston is located during the feed mode of operation, and the hub and the conical piston are cooperatively configured such that the centrate valve is urged to the open position when the conical piston is located against the hub.

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42. The centrifugal separator of claim 41, wherein the conical piston includes one or more springs operative to bias the centrate valve in the closed position.

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43. The centrifugal separator of claim 42, wherein the hub includes one or more downward-extending pins, and the conical

piston includes one or more openings through which the pins extend when the conical piston is disposed adjacent the hub, the pins contacting the centrate valve and maintaining it in the open position when the conical piston is located adjacent the hub.

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44. The centrifugal separator of claim 34, wherein the coupling between the conical piston and the piston actuator is operative to permit the piston to be held at an uppermost position in the bowl by hydraulic pressure from the feed liquid in the feed mode of operation.

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45. The centrifugal separator of claim 34, wherein the coupling between the conical piston and the piston actuator comprises:

a lower piston shaft extending upwardly from the piston and having a hollow upper portion; and

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a hollow upper piston shaft having first and second positions with respect to the lower piston shaft, the first position being a disconnected position in which the upper piston shaft is withdrawn from the hollow portion of the lower piston shaft, the second position being a connected position in which the upper piston shaft is inserted within the hollow portion of the lower piston shaft and mechanically linked thereto.

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46. The centrifugal separator of claim 45, wherein the upper piston shaft including a plurality of flexible, downward-extending fingers that in a relaxed position permit the upper piston shaft to slide with respect to the lower piston shaft.

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47. The centrifugal separator of claim 45, wherein a coupling lock draw bar is disposed within the upper piston shaft, the coupling lock draw bar being configured at a lower end thereof to urge the fingers of the upper piston shaft outwardly against the internal wall of the lower piston shaft to lock the upper and

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lower piston shafts together when the coupling lock draw bar is urged upwardly with respect to the upper piston shaft.

5 48. The centrifugal separator of claim 45, wherein a mechanism at the upper end of the upper piston shaft is operative to selectively apply an upward force on the coupling lock draw bar with respect to the upper piston shaft to mechanically link the upper and lower piston shafts together.

10 49. The centrifugal separator of claim 45, wherein the force-applying mechanism comprises a coupling lock cylinder through which the coupling lock draw bar extends, the coupling lock cylinder including an actuatable piston operative in response to an actuating force to apply the upward force to the coupling  
15 lock draw bar.

50. The centrifugal separator of claim 46, wherein the actuating force is hydraulic or pneumatic.

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## AMENDED CLAIMS

[received by the International Bureau on 03 March 2006 (03.03.2006) :  
original claims 15-32 cancelled ; remaining claims unchanged]

What is claimed is:

1. A divert assembly for a centrifugal separator, comprising:  
a first valve member proximate to an opening of a separator bowl, the first valve member operatively coupleable to a valve actuator for rotating the first valve member about a rotational axis;  
a second valve member cooperating with a lower surface of the first valve member when the first valve member is in a closed position; and  
a valve piston having an uppermost end at which the second valve member is proximately disposed, the valve piston operative to move the second valve member with respect to the bowl.
2. The divert assembly for a centrifugal separator of claim 1, wherein during a solids pumping mode of operation the valve piston moves the second valve member upward along a vertical axis to cooperate with the opening in the bowl.
3. The divert assembly for a centrifugal separator of claim 1, wherein during a feed mode of operation the first valve member is in the closed position permitting feed liquid to be injected into the bowl.
4. The divert assembly for a centrifugal separator of claim 3, wherein the first valve member defines a feed passage, the feed passage cooperating with the opening in the bowl during the feed mode of operation to permit feed liquid to be injected into the bowl.
5. The divert assembly for a centrifugal separator of claim 1, wherein the first valve member defines a drain passage, the drain

passage operative to permit centrate to drain from the opening in the bowl when the first valve member is in the closed position.

6. The divert assembly for a centrifugal separator of claim 5, wherein the opening in the bowl and the drain passage are configurable to enable centrate to drain by gravity from the bowl into the drain passage.

7. The divert assembly for a centrifugal separator of claim 1, wherein a first passage is partially disposed within the valve piston, the first passage cooperating with the second valve member at the uppermost end of the valve piston.

8. The divert assembly for a centrifugal separator of claim 7, wherein the first passage extends beyond a lowermost end of the valve piston.

9. The divert assembly for a centrifugal separator of claim 7, wherein the opening in the bowl and the first passage are configurable to enable solids from the bowl to pass through the first passage during a solids pumping mode of operation.

10. The divert assembly for a centrifugal separator of claim 7, wherein the first passage cooperates with a second passage partially disposed within the valve piston.

11. The divert assembly for a centrifugal separator of claim 10, wherein the second passage extends beyond a lowermost end of the valve piston.

12. The divert assembly for a centrifugal separator of claim 11, wherein compressed air or hydraulic fluid introduced through a

port for the second passage enters the first passage to contact solids therein.

13. The divert assembly for a centrifugal separator of claim 1, wherein an annular flange is disposed about the valve piston.

14. The divert assembly for a centrifugal separator of claim 13, wherein the valve piston moves in response to compressed air or hydraulic fluid contacting the annular flange.

15. (Cancelled)

16. (Cancelled)

17. (Cancelled)

18. (Cancelled)

19. (Cancelled)

20. (Cancelled)

21. (Cancelled)

22. (Cancelled)

23. (Cancelled)

24. (Cancelled)

25. (Cancelled)

26. (Cancelled)

27. (Cancelled)

28. (Cancelled)

29. (Cancelled)

30. (Cancelled)

31. (Cancelled)

32. (Cancelled)

33. A centrifugal separator, comprising:

a cylindrical bowl having a frustoconical lower end with an opening, the bowl being operative during a feed mode of operation to rotate at a high speed to separate the feed liquid into centrate and solids, the solids accumulating along an inner surface of the bowl;

a piston assembly including a conical piston coupled to a piston actuator, the piston being movably disposed against the inner surface of the bowl, the piston actuator operative to move the piston along a vertical axis to force accumulated solids from the opening in the bowl;

a first valve assembly, wherein the first valve assembly comprises

a first valve member, the first valve member proximate to an opening of a separator bowl, and

an actuator operatively coupleable to the first valve member for rotating the first valve member about a rotational axis; and

a second valve assembly, wherein the second valve assembly comprises



a second valve member, the second valve member cooperating with the first valve member of the first valve assembly when the first valve member is in a closed position, and

a valve piston, the valve piston having an uppermost end at which the second valve member is proximately disposed, wherein the valve piston is operative to move the second valve member with respect to the bowl.

34. The centrifugal separator of claim 33, wherein during a solids pumping mode of operation the valve piston moves the second valve member upward along a vertical axis to cooperate with the opening in the bowl.

35. The centrifugal separator of claim 33, wherein during a feed mode of operation the first valve member is in the closed position permitting feed liquid to be injected into the bowl.

36. The centrifugal separator of claim 35, wherein the first valve member defines a feed passage, the feed passage cooperating with the opening in the bowl during the feed mode of operation to permit feed liquid to be injected into the bowl.

37. The centrifugal separator of claim 33, wherein the first valve member defines a drain passage, the drain passage operative to permit centrate to drain from the opening in the bowl when the first valve member is in the closed position.

38. The centrifugal separator of claim 37, wherein the opening in the bowl and the drain passage are configurable to enable centrate to drain by gravity from the bowl into the drain passage.

39. The centrifugal separator of claim 34, wherein the conical piston includes a centrate valve having an open position in the

feed mode of operation and a closed position in the solids pumping mode of operation, the centrate valve being operative in the open position to permit the flow of the centrate from the bowl into a passage to a centrate discharge port of the separator, the centrate valve being operative in the closed position to block the passage of the accumulated solids from the bowl into the passage to the centrate discharge port.

40. The centrifugal separator of claim 39, wherein the centrate valve is an annular member located at substantially the midsection of the conical piston.

41. The centrifugal separator of claim 39, wherein the conical piston is configured to bias the centrate valve in the closed position, the bowl includes a hub against which the conical piston is located during the feed mode of operation, and the hub and the conical piston are cooperatively configured such that the centrate valve is urged to the open position when the conical piston is located against the hub.

42. The centrifugal separator of claim 41, wherein the conical piston includes one or more springs operative to bias the centrate valve in the closed position.

43. The centrifugal separator of claim 42, wherein the hub includes one or more downward-extending pins, and the conical piston includes one or more openings through which the pins extend when the conical piston is disposed adjacent the hub, the pins contacting the centrate valve and maintaining it in the open position when the conical piston is located adjacent the hub.

44. The centrifugal separator of claim 34, wherein the coupling between the conical piston and the piston actuator is operative to

permit the piston to be held at an uppermost position in the bowl by hydraulic pressure from the feed liquid in the feed mode of operation.

45. The centrifugal separator of claim 34, wherein the coupling between the conical piston and the piston actuator comprises:

a lower piston shaft extending upwardly from the piston and having a hollow upper portion; and

a hollow upper piston shaft having first and second positions with respect to the lower piston shaft, the first position being a disconnected position in which the upper piston shaft is withdrawn from the hollow portion of the lower piston shaft, the second position being a connected position in which the upper piston shaft is inserted within the hollow portion of the lower piston shaft and mechanically linked thereto.

46. The centrifugal separator of claim 45, wherein the upper piston shaft including a plurality of flexible, downward-extending fingers that in a relaxed position permit the upper piston shaft to slide with respect to the lower piston shaft.

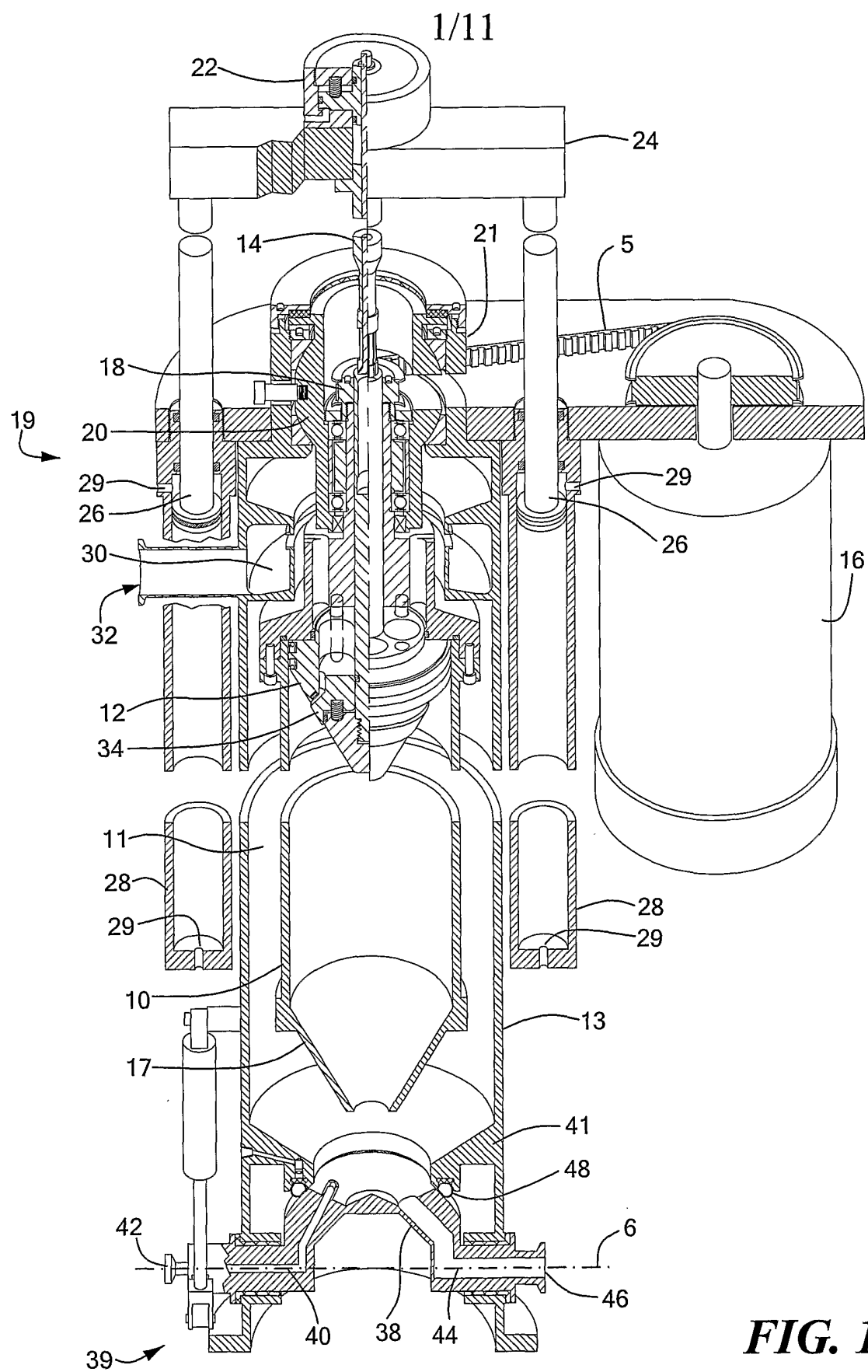
47. The centrifugal separator of claim 45, wherein a coupling lock draw bar is disposed within the upper piston shaft, the coupling lock draw bar being configured at a lower end thereof to urge the fingers of the upper piston shaft outwardly against the internal wall of the lower piston shaft to lock the upper and lower piston shafts together when the coupling lock draw bar is urged upwardly with respect to the upper piston shaft.

48. The centrifugal separator of claim 45, wherein a mechanism at the upper end of the upper piston shaft is operative to selectively apply an upward force on the coupling lock draw bar

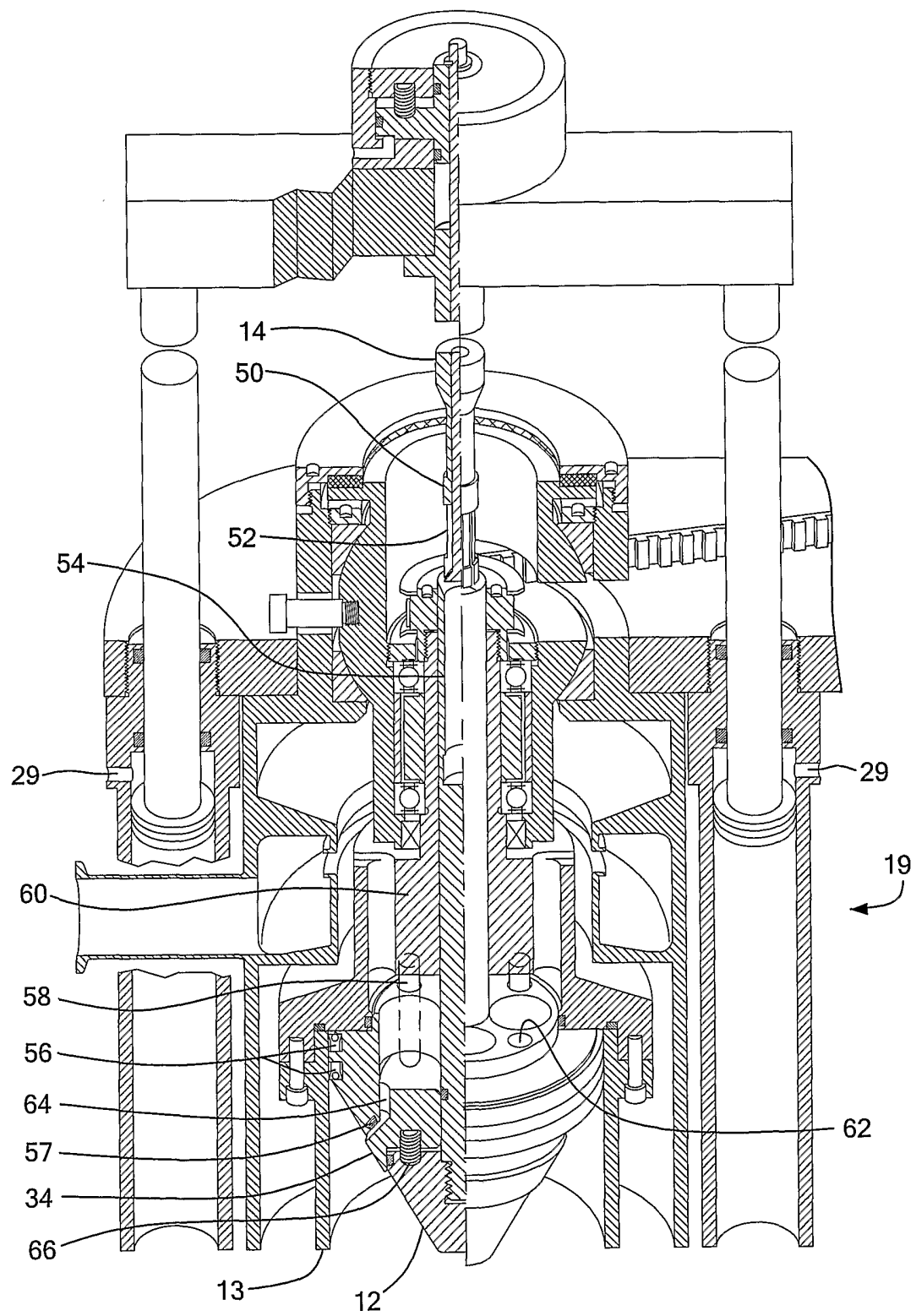
with respect to the upper piston shaft to mechanically link the upper and lower piston shafts together.

49. The centrifugal separator of claim 45, wherein the force-applying mechanism comprises a coupling lock cylinder through which the coupling lock draw bar extends, the coupling lock cylinder including an actuatable piston operative in response to an actuating force to apply the upward force to the coupling lock draw bar.

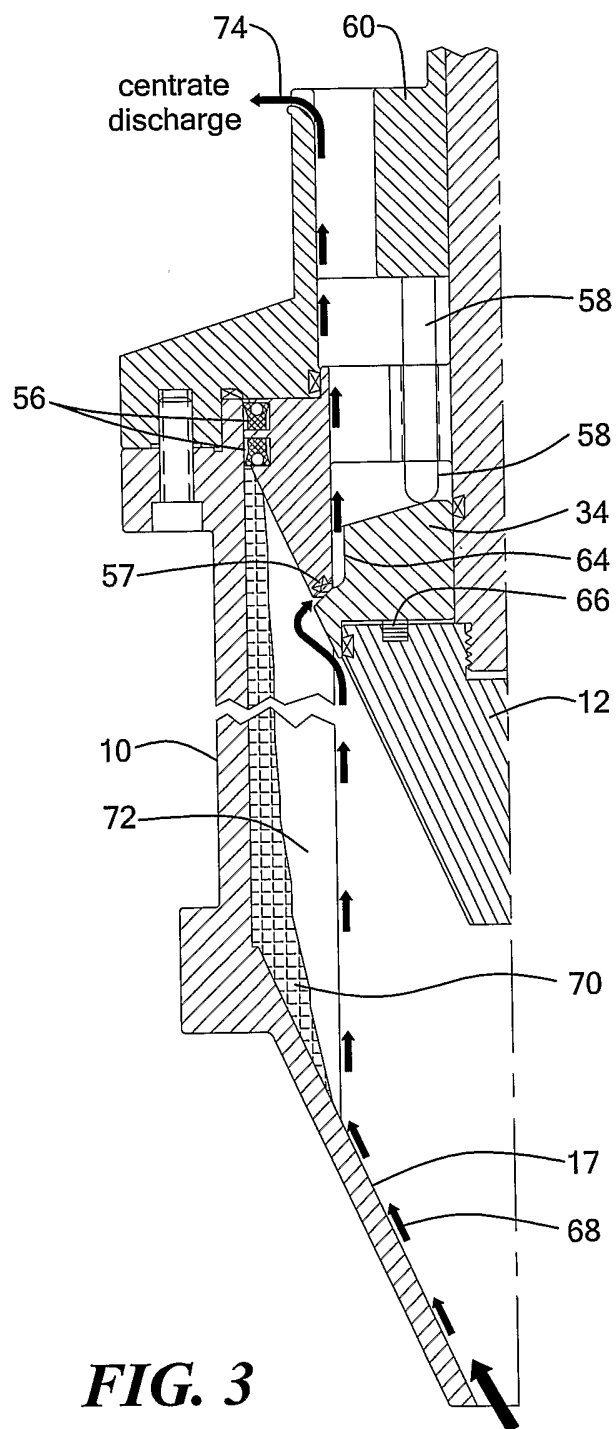
50. The centrifugal separator of claim 46, wherein the actuating force is hydraulic or pneumatic.



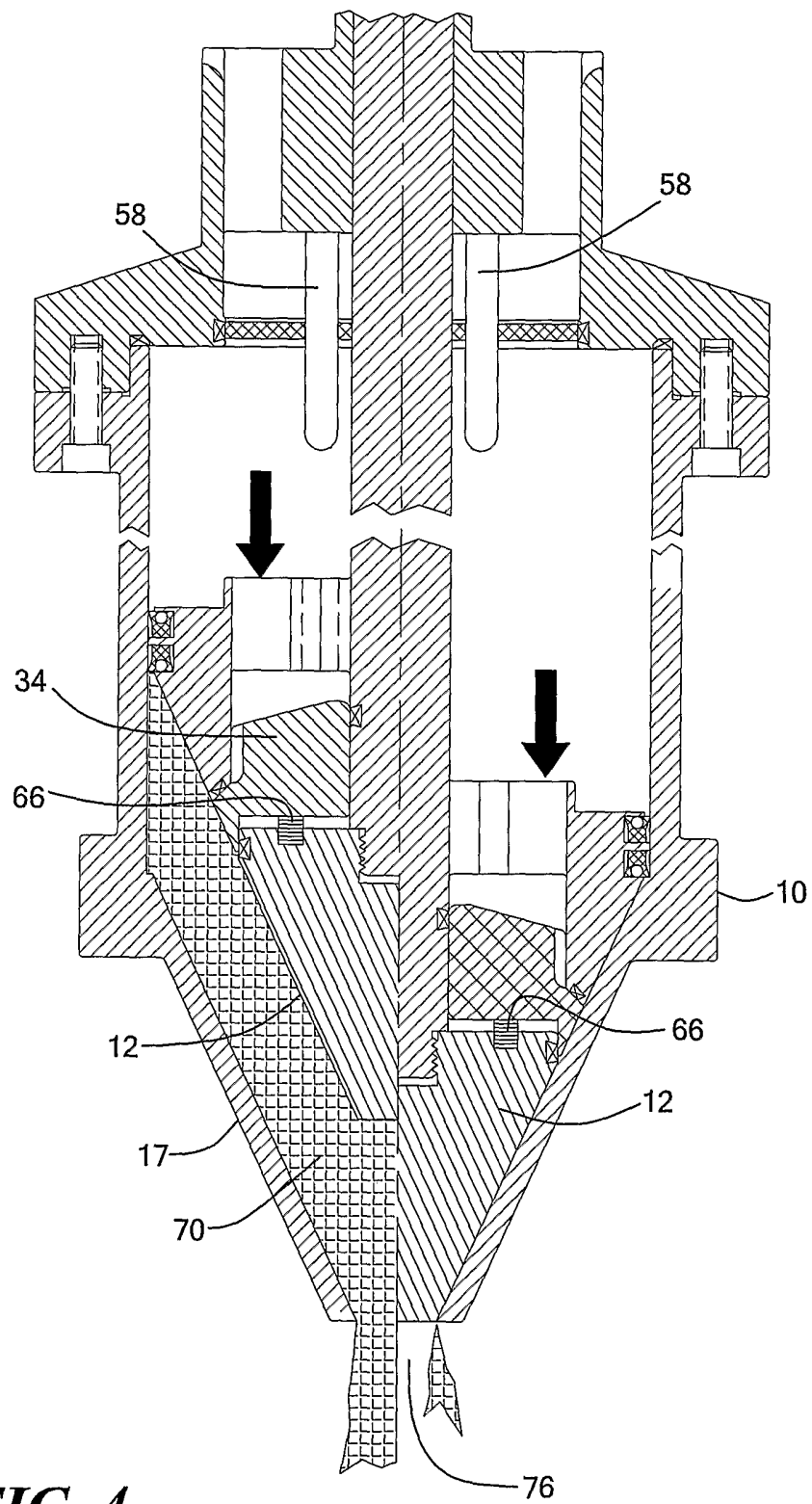
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**FIG. 2**

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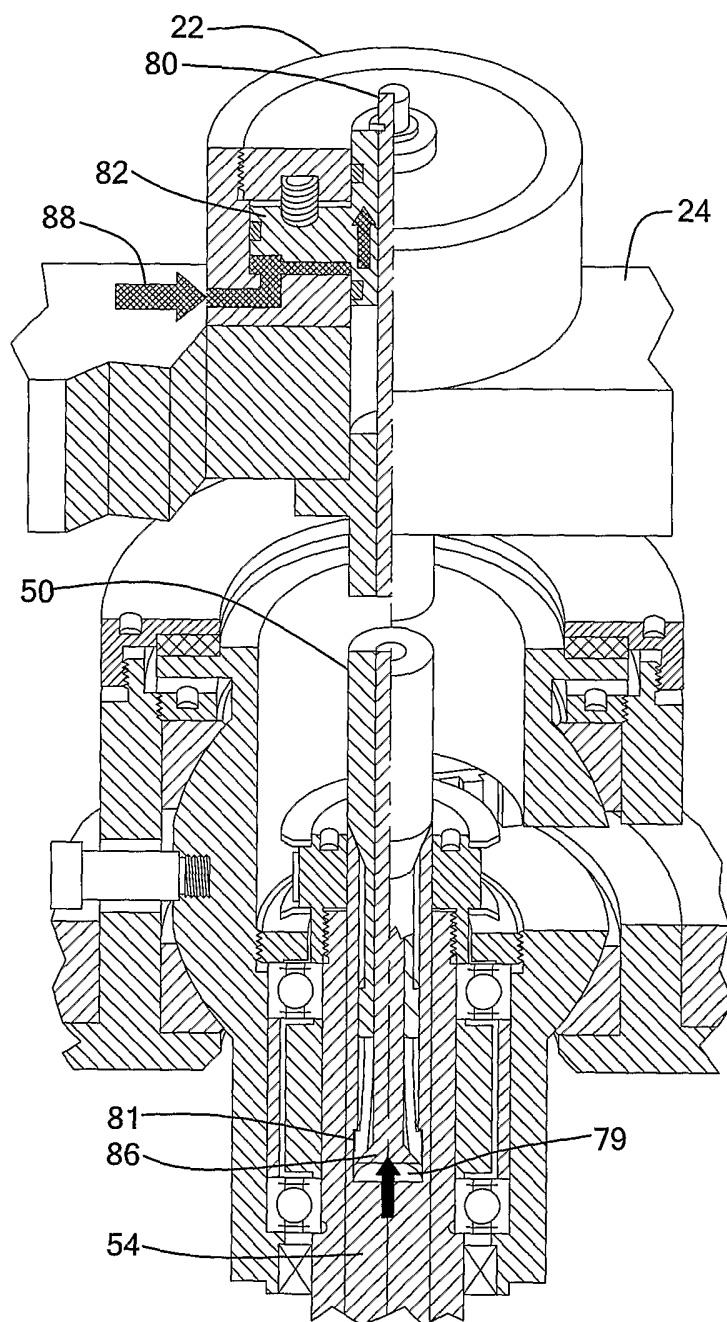
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**FIG. 4**

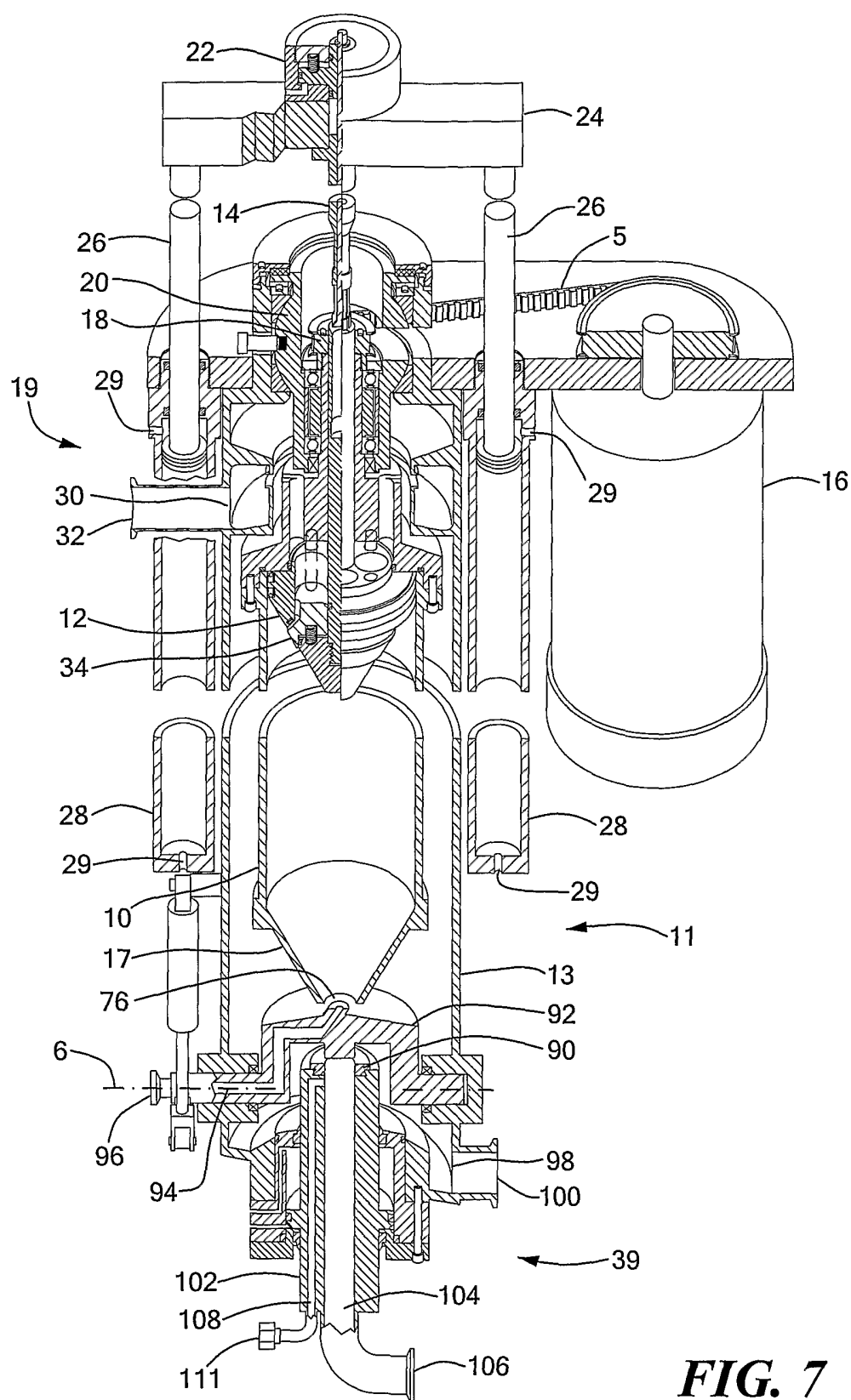




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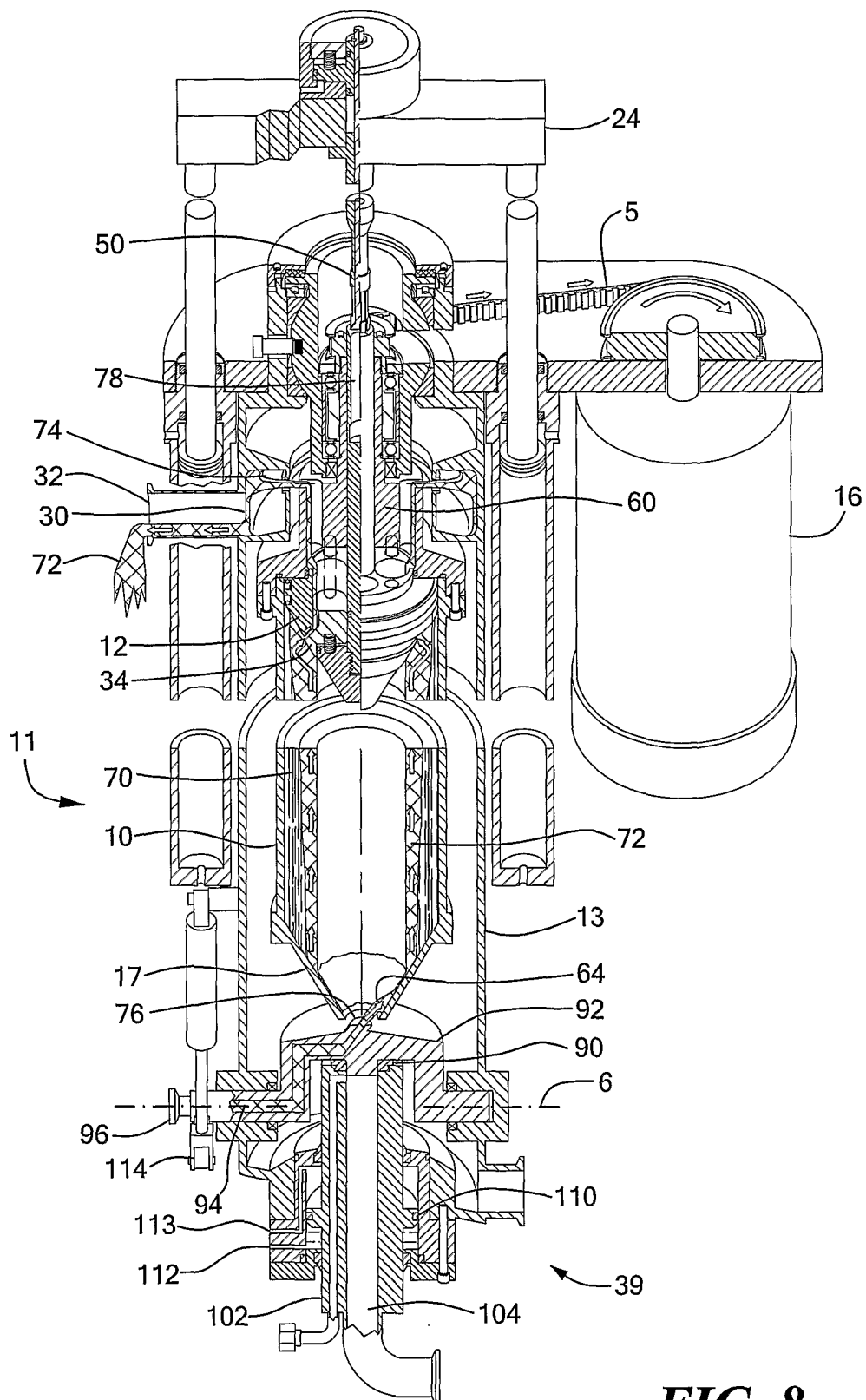
**FIG. 6**

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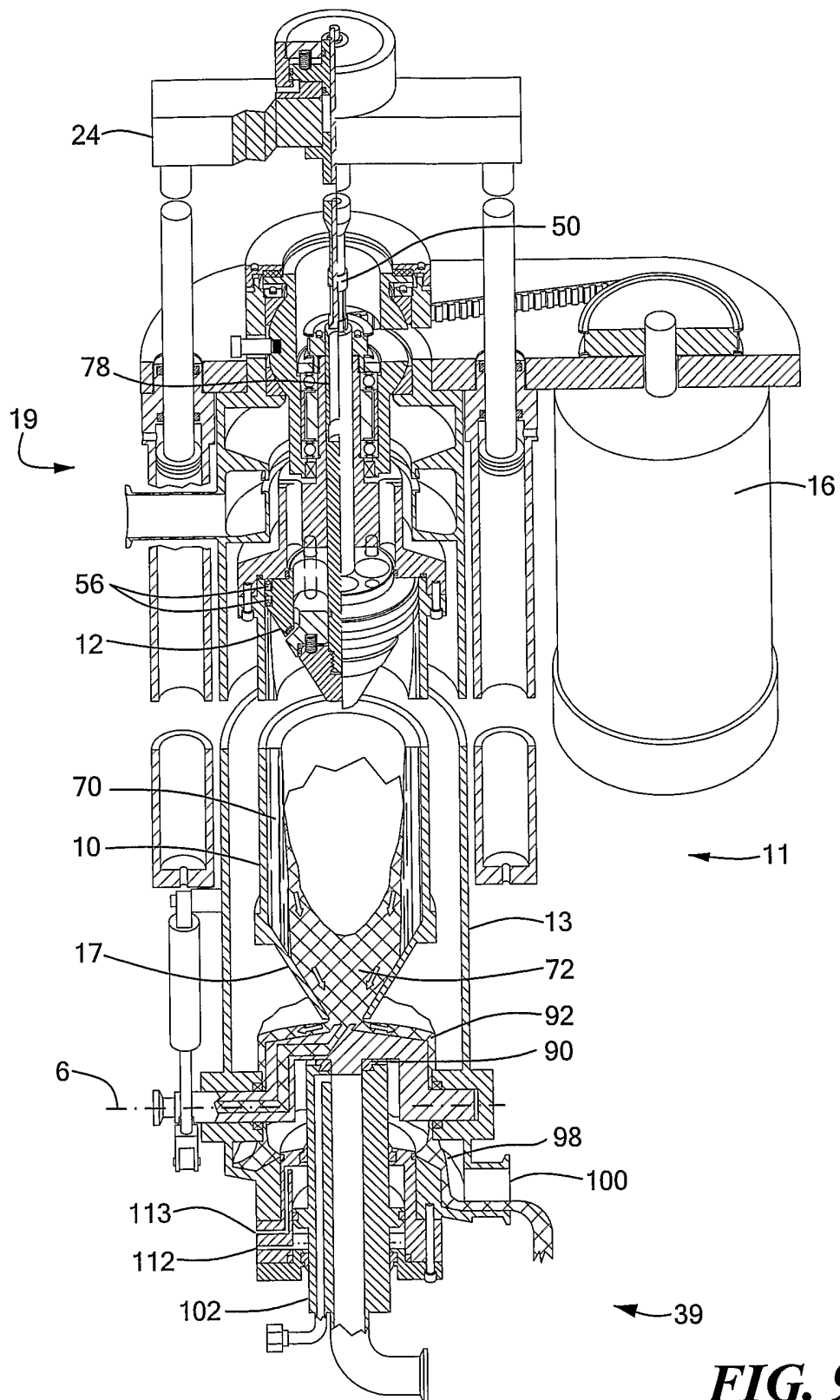


**FIG. 7**

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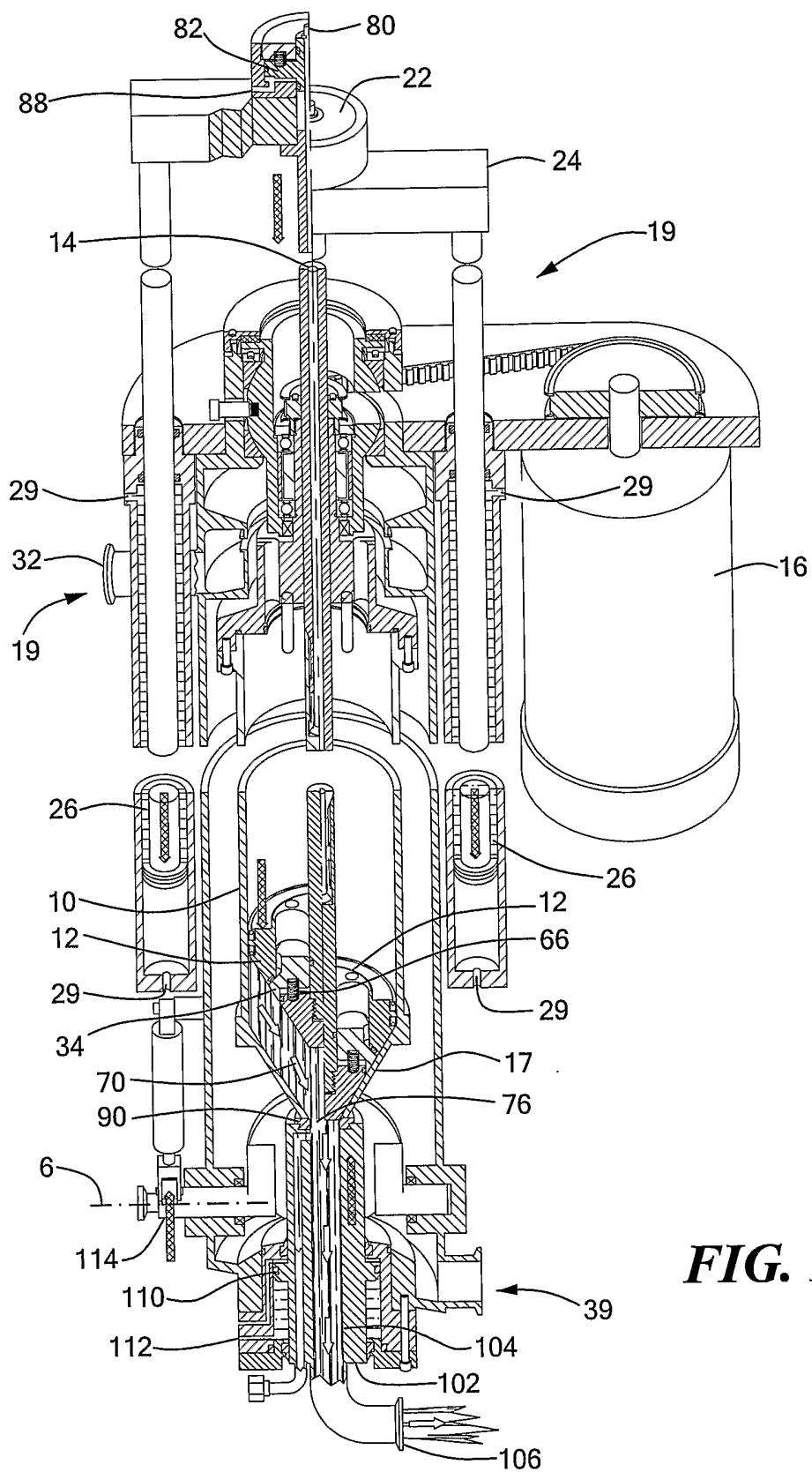
**FIG. 8**

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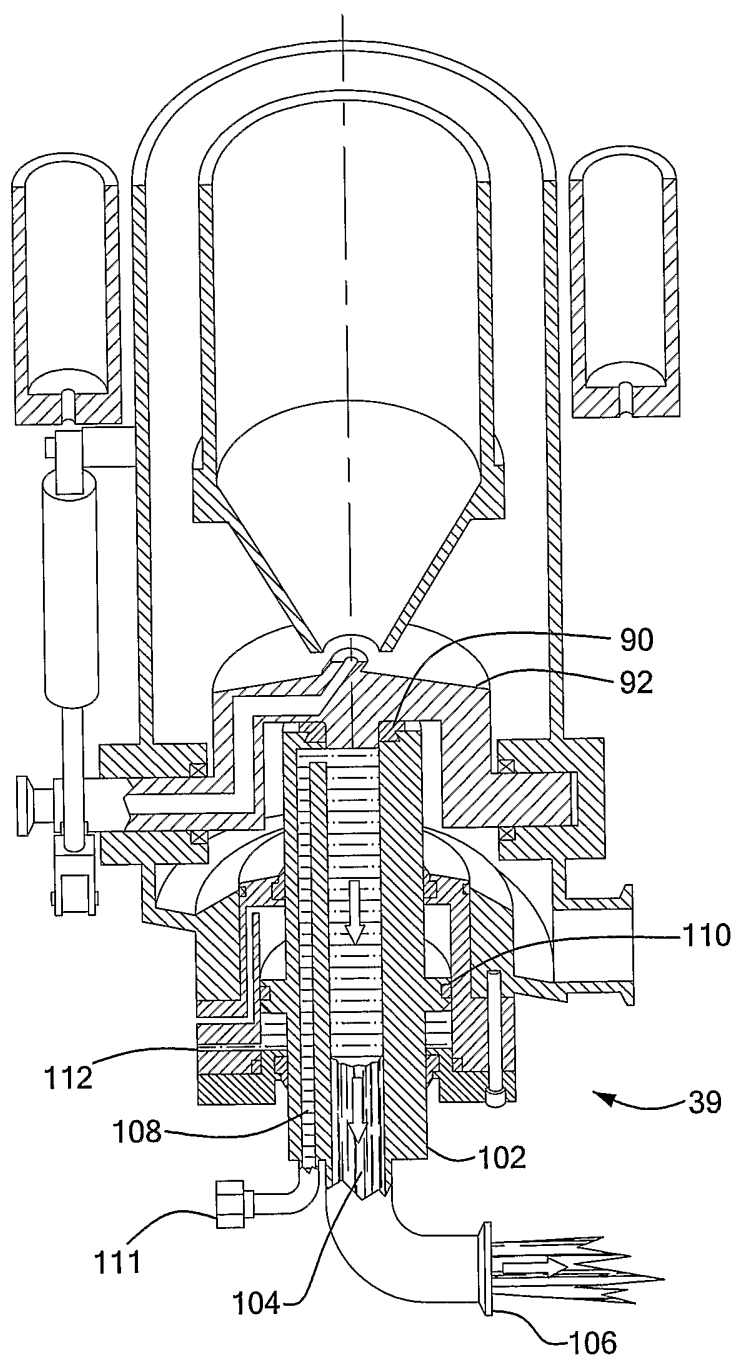


**FIG. 9**


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**FIG. 10**

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**FIG. 11**

# INTERNATIONAL SEARCH REPORT

Inter:  nual Application No  
PCT/IB2005/003457

## A. CLASSIFICATION OF SUBJECT MATTER

B04B11/05 B04B15/06

According to International Patent Classification (IPC) or to both national classification and IPC

## B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

B04B

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

EPO-Internal, WPI Data, PAJ

## C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category *	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
A	WO 03/086641 A (R.B. CARR ENGINEERING, INC) 23 October 2003 (2003-10-23) the whole document -----	1-50
A	US 2002/016243 A1 (CARR ROBERT B) 7 February 2002 (2002-02-07) the whole document -----	1-50
A	WO 00/38762 A (BIOSAFE S.A; FELL, CLAUDE) 6 July 2000 (2000-07-06) the whole document -----	1-50

☐ Further documents are listed in the continuation of box C.

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Date of the actual completion of the international search

20 December 2005

Date of mailing of the International search report

11/01/2006

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International Application No  
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WO 0038762 A	06-07-2000	AT 271890 T AU 1675600 A CN 1331610 A DE 69919029 D1 DE 69919029 T2 EP 1144026 A1 ES 2224726 T3 HK 1041454 A1 JP 2002533171 T PT 1144026 T US 6733433 B1	15-08-2004 31-07-2000 16-01-2002 02-09-2004 08-09-2005 17-10-2001 01-03-2005 31-12-2004 08-10-2002 31-12-2004 11-05-2004