

Dec. 29, 1970

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3,551,076

TUBULAR DIAPHRAGM PUMP

Filed March 22, 1968

2 Sheets-Sheet 1

FIG. 1

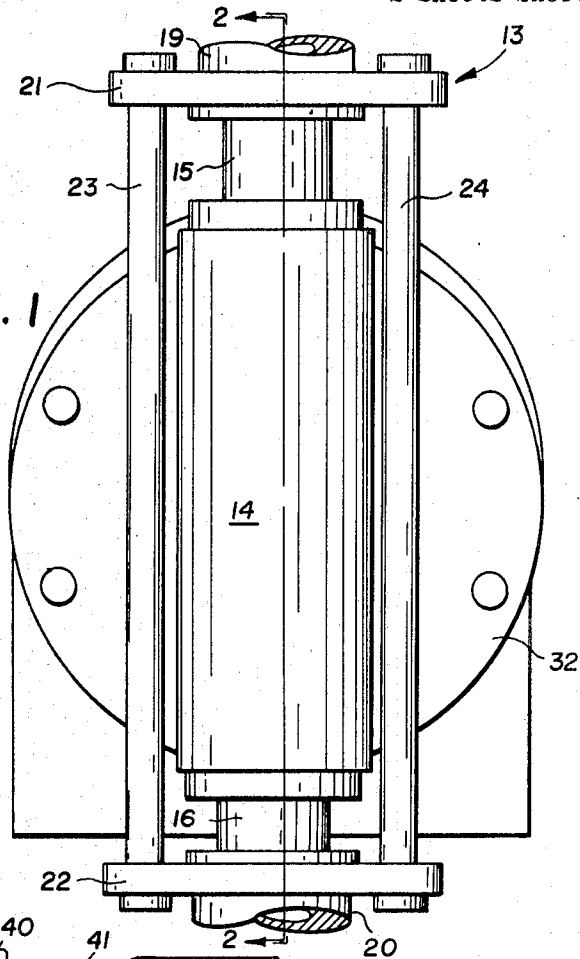
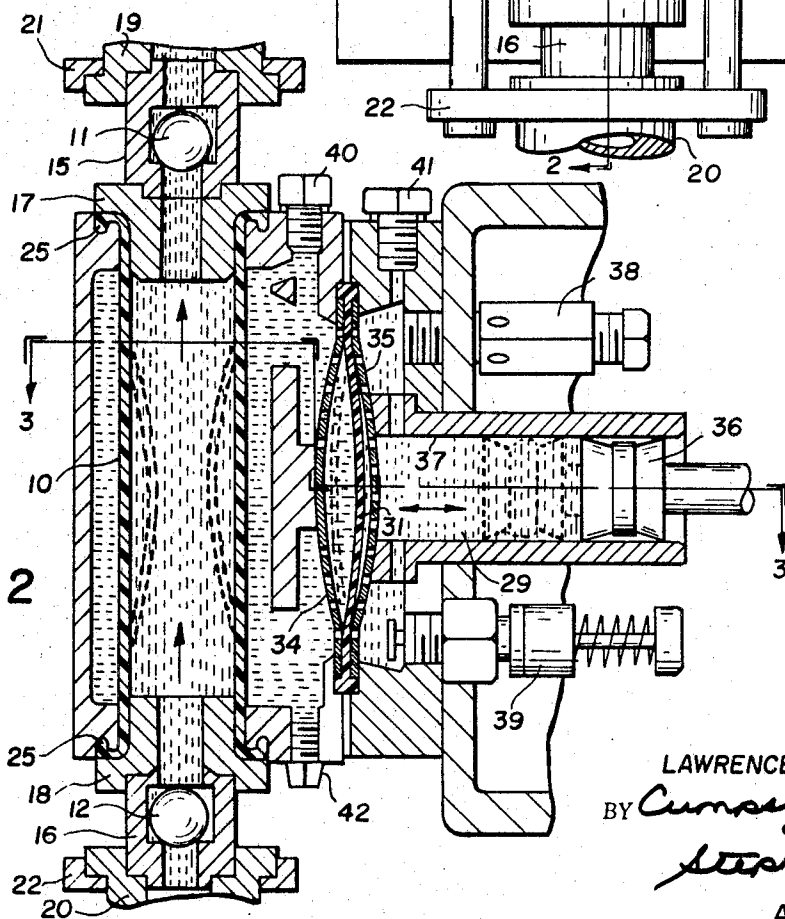


FIG. 2



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2 Sheets-Sheet 2

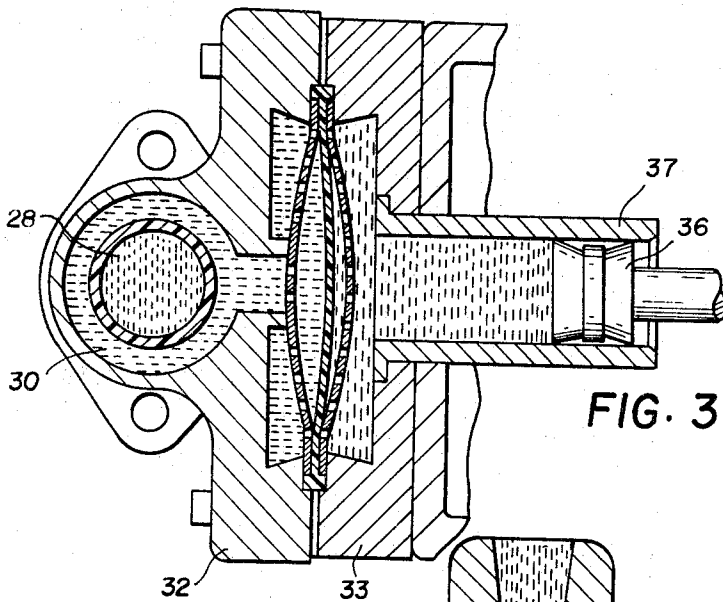


FIG. 3

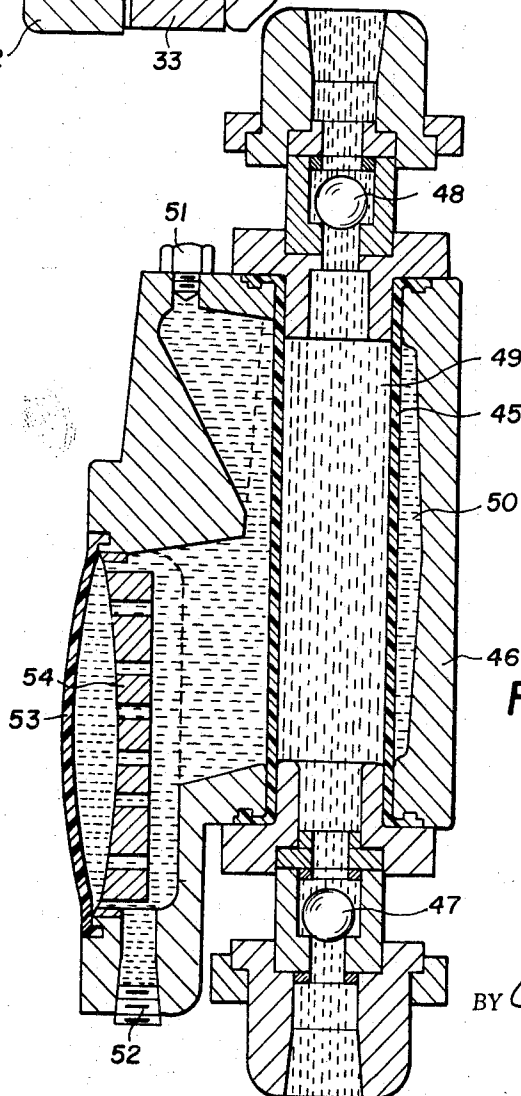


FIG. 4

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TUBULAR DIAPHRAGM PUMP

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U.S. Cl. 417—385

4 Claims

ABSTRACT OF THE DISCLOSURE

A hydraulically reciprocated diaphragm drives a tubular diaphragm through which material is pumped in positive displacements, and a predetermined volume of liquid couples the diaphragm and the tube and surrounds the tube. The diaphragm's excursion is limited, and the liquid volume is set to compress but not expand the tube relative to its normal cylindrical shape.

Diaphragm pumps are widely used in metering and pumping corrosive or abrasive materials which the diaphragm separates from moving pump parts. At the present state of the art, such diaphragm pumps use circular diaphragms that are reciprocated between apertured support plates to draw in volumes of pumped liquid and force such volumes out. Valves limit the motion of the pumped liquid to a single direction so that the diaphragm acts as a positive displacement pump. Two of such diaphragms can be arranged in series with an inner diaphragm driving an outer diaphragm which pumps the liquid. This protects the diaphragm driving mechanism from a diaphragm failure admitting corrosive or abrasive materials past the outer diaphragm.

The disadvantages of such diaphragm pumps are their expense and relatively short life under certain adverse conditions. The support plates for limiting the diaphragm excursion are complex and expensive and expose large surface areas to liquid environments. When the liquid environment is a hostile corrosive or abrasive material being pumped, a large surface area is available for attack, and an expensive part is rapidly deteriorated. Also, abrasives or foreign particles in the pumped liquid can lodge in the diaphragm support plate region and damage or weaken the diaphragm to reciprocate its failure.

The objects of the invention include, without limitation:

(a) Overcoming the disadvantages of prior art diaphragm pumps;

(b) A diaphragm pump that is capable of pumping slurries, abrasives, high viscosity products, or highly corrosive materials with ease, simplicity, and reliability;

(c) A diaphragm pump that allows use of optimum wear materials at each part of the pump;

(d) A diaphragm pump that is simple, reliable, long-lived, and yet highly versatile in the materials and rates it is capable of pumping; and

(e) A diaphragm pump that better protects its expensive moving parts from damage from a diaphragm failure.

These and other objects of the invention will be apparent hereinafter from the specification which describes the invention, its use, operation and preferred embodiment, from the drawings, which constitute a part of the disclosure, and from the subject matter claimed.

Generally, the inventive pump includes: a limited excursion, reciprocated diaphragm; a flexible tube forming a passageway for pumped material; valves requiring a one-way flow of pumped material through the tube; a liquid-tight chamber around the tube and having the diaphragm arranged as a portion of the chamber bound-

ary; and the chamber being completely filled with a predetermined volume of liquid. This results in the diaphragm driving the tube in compressions and re-expansions to normal to serve as a positive displacement pump. Preferably, the volume of liquid is selected so that when diaphragm excursion gives the chamber maximum volume, the tube assumes its normal configuration of a cylinder of uniform diameter, and when diaphragm excursion gives the chamber minimum volume, the tube is compressed.

In the drawings:

FIG. 1 is a front elevation of a preferred embodiment of the inventive pump;

FIG. 2 is a cross-section of the pump of FIG. 1 taken along the line 2—2 thereof;

FIG. 3 is a cross-section of a pump of FIG. 2 taken along the line 3—3 thereof; and

FIG. 4 is a cross-sectional view of a preferred alternative embodiment of the inventive pump.

The pumping diaphragm of the inventive pump illustrated in FIGS. 1-3 is a flexible tube 10 arranged so that the pumped liquid or material 28 (shaded for purple) passes through tube 10. Tube 10 is preferably formed of a flexible material resistant to liquid 28 and having a substantial flex-life. Preferred materials for tube 10 include fluorinated elastomers or fluorinated or polytetrafluorinated polymers such as polytetrafluoroethylene. Ball valves 11 and 12 restrict the flow of liquid 28 to the direction indicated by the arrows in FIG. 2 so that when tube 10 is compressed to its broken line position as shown in FIG. 2, liquid 28 is forced out through outlet valve 11, and when tube 10 is re-expanded to its solid line position, liquid 28 is brought in through inlet valve 12.

Tube 10 and valves 11 and 12 are arranged in a reagent head 13 including a plate 32 forming cylinder 14, valve housings 15 and 16, collars 17 and 18, caps 19 and 20, and connector plates 21 and 22 secured together by bolts 23 and 24. Each end of tube 10 is formed with a peripheral flange 25 that is clamped between cylinder 14 and collars 17 and 18 respectfully to form a liquid-tight seal between tube 10 and the pump housing.

A chamber formed within cylinder 14 surrounds the mid-section of tube 10 and is filled with a coupling liquid 30 (shaded for gray). Tube 10 is flexed and driven through coupling liquid 30 as described below.

A conventional circular diaphragm 31 is secured between head plate 32 and body plate 33 of the pump and has apertured support plates 34 and 35 arranged on its opposite sides for limiting its excursion. Diaphragm 31 forms a portion of the boundary wall of the chamber surrounding tube 10 and containing liquid 30 so that diaphragm 31 communicates with tube 10 through coupling liquid 30. Diaphragm 31 is preferably formed of polyurethane or a similar synthetic resin.

Diaphragm 31 is reciprocated between support plates 34 and 35 by piston 36 that is reciprocated in cylinder 37 as shown by the arrows. A driving liquid 29 (shaded for purple) communicates with piston 36 and diaphragm 31 so that the reciprocating motion of piston 36 impresses a corresponding reciprocating motion on diaphragm 31. Spring-loaded ball valves 38 and 39 are arranged in respective inlet and outlet openings for regulating the amount of liquid 29. Plugs 40, 41, and 42 stop filling and drain holes for the liquids 29 and 30.

Liquid 29 is preferably oil, but can be any suitable hydraulic material. Liquid 30 is preferably compatible with the material 28 being pumped so that minimum damage will be done by any rupture of tube 10. Liquid 30 can be water for many applications. If liquid 30 is properly selected, detectors such as electrical detectors or visual detectors can be used to indicate any leakage of the pumped material 28 into the chamber normally filled by liquid 30.

Tube 10 normally assumes the configuration of a cylinder with uniform diameter such as shown in solid lines in FIG. 2, and the amount of liquid 30 is selected so that tube 10 assumes such normal configuration when diaphragm 31 is in its maximum excursion position as illustrated in solid lines in FIG. 2. This results in repeated compression of tube 10 inward from its normally assumed cylindrical shape, but no expansion of tube 10 outward beyond such cylindrical shape. Since the amount of liquid 30 in the chamber surrounding tube 10 is fixed and cannot increase, such a relationship between tube 10 and diaphragm 31 is readily maintained during pumping.

In the alternative embodiment illustrated in FIG. 4, resilient tube 45 is arranged in a similar fashion in head 46 with ball valves 47 and 48 checking the inlet and outlet respectively for pumped material 49 (shaded for purple) moving through tube 45. The chamber surrounding tube 45 is filled with a coupling liquid 50 (shaded for gray) of a pre-selected volume. Plugs 51 and 52 stop the inlet and outlet openings to the chamber containing liquid 50. Circular diaphragm 53 and one of its support plates 54 are shown in communication with liquid 50 for driving tube 45 in a manner similar to the embodiment of FIGS. 1-3. The differences in the FIG. 4 embodiment are generally a simplified pump housing and reagent head 46 affording more economical construction.

In operating the embodiment of FIGS. 1-3, a power source reciprocates piston 36 in cylinder 37 to make reciprocating displacements in driving liquid 29. These move diaphragm 31 in a reciprocating motion between its support plates 34 and 35. The motion of diaphragm 31 produces displacements in coupling liquid 30 that repeatedly squeeze tube 10 inward to discharge material 28 through outlet valve 11 and then draw tube 10 back to its normal cylindrical configuration to intake liquid 28 in through inlet valve 12. The result is positive displacement in tube 10 that move the pumped liquid 28 in the direction of the arrows in FIG. 2.

The flexure of tube 10 is controlled through its direct relationship with diaphragm 31 whose excursion is limited. Regardless of blockages or other conditions in the line for the pumped material 28. Diaphragm 31 is limited in its travel between support plates 34 and 35. Also, the amount of coupling liquid 30 is predetermined and fixed so that only predetermined displacements are transmitted to tube 10. Hence tube 10 cannot be over-compressed or forced into the outlet, nor can it be over-expanded to be drawn outward beyond its normal cylindrical shape. This protects tube 10 against undue stresses and prolongs its life. Also, it allows the inventive pump to be used safely on high viscosity materials and in circumstances in which line pressures fluctuate widely.

Tube 45 of the embodiment of FIG. 4 is subjected to a controlled flexure in a similar manner by diaphragm 53.

Only the valves and a minimum of surfaces of the inventive pump are exposed to the pumped liquid, and deterioration of parts from corrosive or abrasive materials in the pumped liquid is minimal. A simple reagent head mounts tubes 10 or 45, and since their flexure is controlled, they can be formed of flexible materials that are highly resistant to the pumped material and still maintain a long life. The large surface areas and complex configurations of support surfaces 54 or 34 do not contact corrosive materials and are not worn excessively during

pumping. Furthermore, eliminating a set of such complex and expensive support surfaces materially reduces the expense of the inventive pump. Conventional diaphragms 31 and 53 can be formed of materials most suited to moving between support surfaces, for example, polyurethane. Such materials are not also the most resistant to corrosive materials that must be pumped. The optimum material such as polytetrafluoroethylene for tubes 10 and 45 is not deteriorated by the pumped material, but is not satisfactory as a conventional diaphragm, because cold flow against support surfaces soon deteriorates and destroys a conventional circular diaphragm made of materials like polytetrafluoroethylene. Hence, the inventive pump permits use of optimum materials for each function within the pump.

Thus, it can be seen that the invention accomplishes its objects in providing a reliable, versatile, and long-lived pump capable of pumping and metering corrosive and abrasive materials.

I claim:

1. A pump comprising:

- (a) a reciprocating piston;
 - (b) a diaphragm;
 - (c) aperture plates arranged on each side diaphragm to limit its excursions;
 - (d) means enclosing a volume of driving liquid between said piston and said diaphragm;
 - (e) said enclosing means having an inlet and an outlet;
 - (f) a valve arranged in said inlet to regulate the inflow of said driving liquid;
 - (g) a valve arranged in said outlet to regulate the outflow of said driving liquid;
 - (h) a flexible tube configured as a cylinder of uniform diameter and arranged to form a passageway through said tube for material to be pumped;
 - (i) valve means for limiting the flow of said material through said tube to a pre-determined direction;
 - (j) means for forming a sealed and liquid-tight chamber around said tube;
 - (k) a portion of the boundary wall of said chamber being formed of said diaphragm; and
 - (l) said chamber being substantially completely filled with a pre-determined volume of liquid so that said tube normally assumes said cylindrical configuration when said excursion of said diaphragm gives said chamber maximum volume.
2. The pump of claim 1 including flanges on the ends of said tube and means for clamping said flanges against said chamber forming means.
3. The pump of claim 1 wherein said tube is formed of a polytetrafluorinated polymer.
4. The pump of claim 1 wherein said tube is formed of a fluorinated elastomer.

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ROBERT M. WALKER, Primary Examiner

U.S. Cl. X.R.

92-90, 98; 417-478

REEXAMINATION CERTIFICATE (162nd)

United States Patent [19]

[11] **B1 3,551,076**

Wilson

[45] **Certificate Issued Feb. 14, 1984**

[54] **TUBULAR DIAPHRAGM PUMP**

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Reexamination Request:
No. 90/000,255, Sep. 9, 1982

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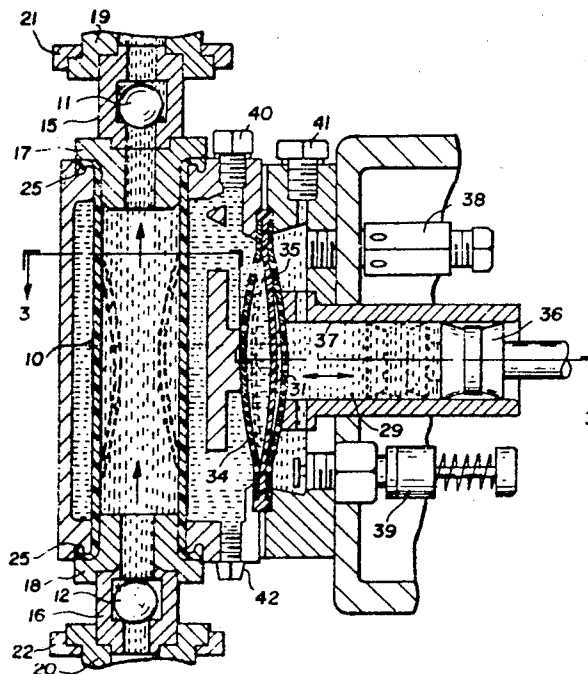
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[57] **ABSTRACT**

A hydraulically reciprocated diaphragm drives a tubular diaphragm through which material is pumped in positive displacements, and a predetermined volume of liquid couples the diaphragm and the tube and surrounds the tube. The diaphragm's excursion is limited, and the liquid volume is set to compress but not expand the tube relative to its normal cylindrical shape.

- [51] **Int. Cl.³** F04B 35/02
- [52] **U.S. Cl.** 417/385; 92/90;
92/98; 417/478
- [58] **Field of Search** 417/389, 478, DIG. 1;
128/1 D; 92/103 SD



**REEXAMINATION CERTIFICATE
ISSUED UNDER 35 U.S.C. 307.**

AS A RESULT OF REEXAMINATION, IT HAS
BEEN DETERMINED THAT:

THE PATENT IS HEREBY AMENDED AS
INDICATED BELOW

Matter enclosed in heavy brackets **[]** appeared in the
patent, but has been deleted and is no longer a part of the
patent; matter printed in italics indicates additions made
to the patent.

Claims 1-4, having been finally determined to be
unpatentable, are cancelled.

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